

## Problem- Investigation of Thermal and Hydrodynamic Boundary Layers During Flow Across a Heated Plate Through ANSYS Simulation

$$T_{\infty} = 300 \text{ K},$$

$$\rho = 1.225 \text{ kg / m}^3$$

$$\mu = 1.7894 \times 10^{-5} \text{ kg / m} \cdot \text{s}$$

$$k = 0.0242 \text{ W / m} \cdot \text{K}$$

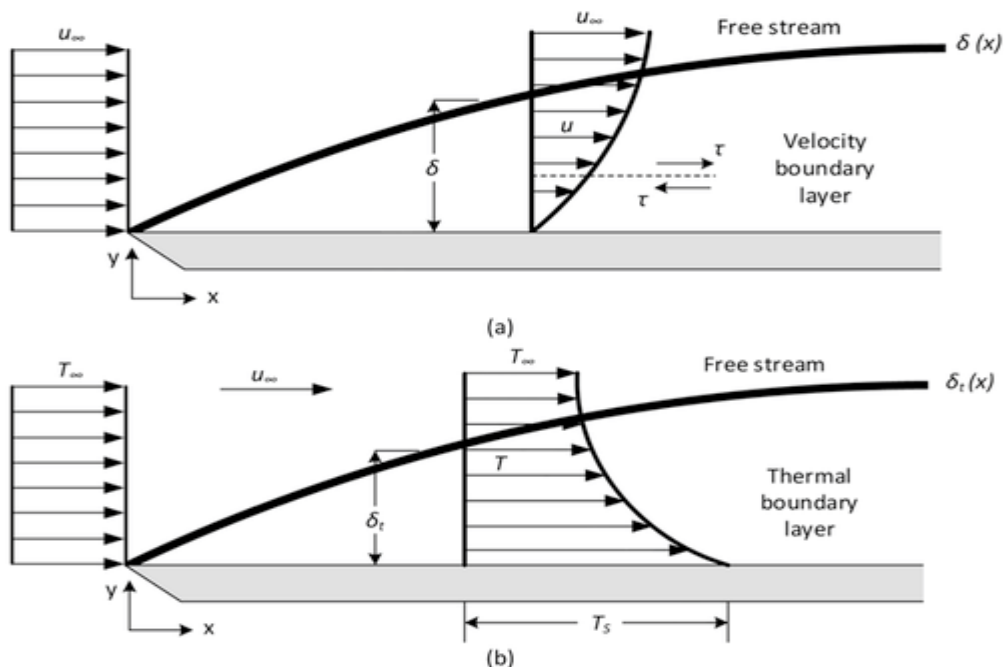
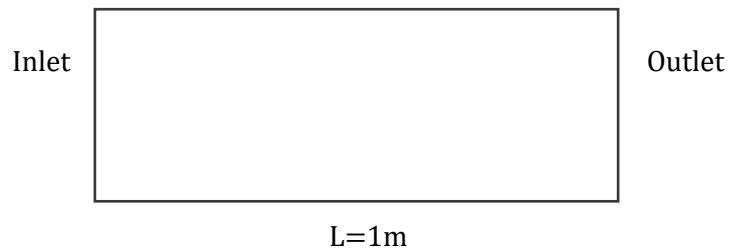
$$c_p = 1006.43 \text{ J / kg} \cdot \text{K}$$

$$Pr = 0.744176$$

$$u_{\infty} = 1.4607 \text{ m / s}$$

$$T_s = \text{Surface temp.} = 400 \text{ K}$$

$$\text{Length of plate} = 1 \text{ m}$$



Theoretical calculation

$$\text{Reynolds number} = Re_L = \rho \cdot u_{\infty} \cdot L / \mu = 1.225 \times 1.4607 \times 1 / 1.7894 \times 10^{-5} = 99998 < 5 \times 10^5$$

Mass flow rate inlet:

$$= u_{\infty} * A = 1.4607 * 0.06 = 0.087642 \approx 0.1 \text{ kg/s}$$

$$\text{Average Nusselt number} = 0.664 Re_L^{1/2} Pr^{1/3} = 0.664 \times 99998^{1/2} \times 0.744176^{1/3} = 190.278$$

$$\text{Average convection heat transfer coefficient} = (\text{Average Nusselt number}) * k / L = 190.278 * 0.0242 = 4.605 \text{ W/m}^2 \cdot \text{K}$$

$$\text{Average skin or fanning friction coefficient} = 1.328 * Re_L^{-1/2}$$

$$= 0.004199547$$

Locally, at  $x = 0.1 \text{ m}$ :

$$Nu_x = 0.332 Re_x^{1/2} Pr^{1/3}$$

$$h_x = 7.28070 \text{ W/m}^2 \cdot \text{K}$$

$$Cf_x = 0.664 Re_x^{-1/2} = 0.006640079$$

$\delta$  = velocity boundary layer thickness

$$\delta = 5L / Re_L^{(1/2)} = 0.01581 \text{ m}$$

$\delta_t$  = Thermal boundary layer thickness

$$\delta / \delta_t = Pr^{1/3}; \quad \delta_t = 0.017448172 \text{ m}$$

## Results-

