Delft University of Technology

Faculty of Aerospace Engineering
Rotor / wake Aerodynamics - AE4135

Rotor / wake Aerodynamics
Assignment 1: Blade Element Theory

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Nomenlacture 1

α	angle of attack at blade element $(-)$
β	blade twist angle at blade element $(-)$
λ	tip speed ratio $(-)$
ρ	fluid density $(\mathrm{kg} \cdot \mathrm{m}^{-3})$
Γ	circulation at blade element $(\mathrm{m}^2 \cdot \mathrm{s}^{-1})$
Φ	perceived-wind inflow-angle at blade element $(-)$
Ω	rotor rotational velocity $(\mathrm{rad} \cdot \mathrm{s}^{-1})$
a	axial induction factor $(-)$
a'	azimuthal induction factor $(-)$
c	blade element chord (m)
C_d	drag coefficient $(-)$
C_l	lift coefficient $(-)$
C_T	thrust coefficient $(-)$
Drag	drag force per unit span $(\mathrm{N} \cdot \mathrm{m}^{-1})$
Lift	lift force per unit span $(\mathrm{N} \cdot \mathrm{m}^{-1})$
F_{azim}	azimuthal/tangential force per unit span $(\mathrm{N} \cdot \mathrm{m}^{-1})$
F_{axial}	axial force per unit span $(\mathrm{N} \cdot \mathrm{m}^{-1})$
N_{blades}	number of blades $(-)$
$V_{\text{axial}} = U_{\text{rotor}}$	axial velocity perceived by blade element, axial velocity at rotor $(\mathrm{m} \cdot \mathrm{s}^{-1})$
V_P	velocity perceived by blade element $(\mathrm{m} \cdot \mathrm{s}^{-1})$
V_{tan}	azimuthal/tangential velocity perceived by blade element $(\mathrm{m} \cdot \mathrm{s}^{-1})$

Introduction 2

Table 2.1: Wind turbine geometrical specifications

Variable	Value
Radius (R)	50 [m]
Number of Blades	3
Blade starts at	0.2 r/R
Twist	14*(1-r/R) [degrees]
Blade Pitch	-2 [degrees]
Chord Distribution	3*(1-r/R)+1 [m]
Airfoil	DU 95-W-180
Rotor yaw angle	0, 15 and 30 [degrees]

Table 2.2: Wind turbine operational specifications

Variable	Value
Wind speed (U0)	10 [m/s]
Tip speed ratio (λ)	6, 8, 10
Rotor yaw angle	0, 15, 30 [degrees]

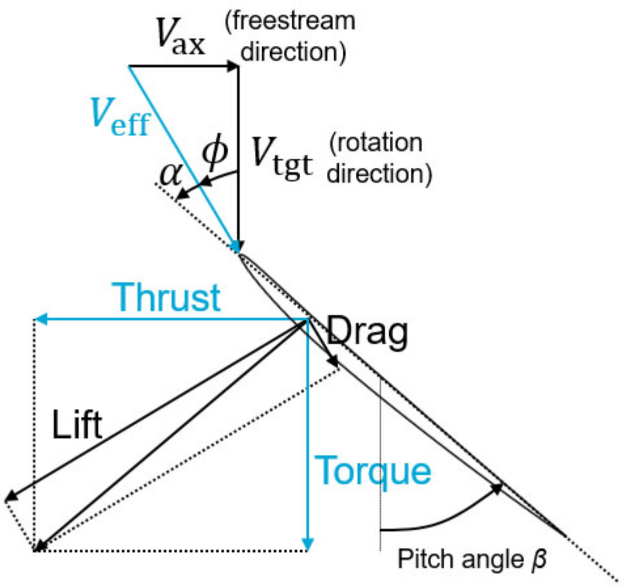

Figure 2.1: Caption

Flow diagram of the code 3

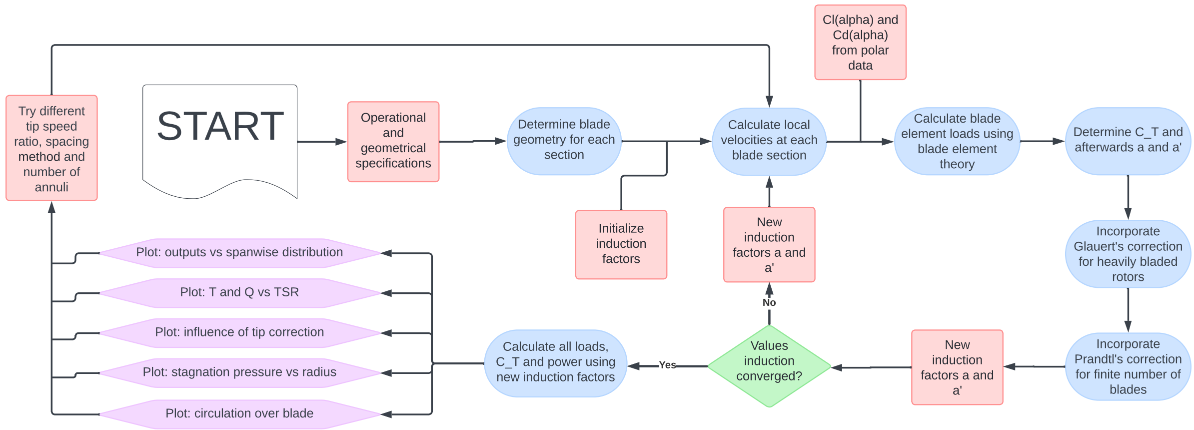

Figure 3.1: Flow diagram of the code

3.1 Assumptions

- **Steady Flow:** the flow characteristics are assumed to be independent of time.
- **Inviscid Flow:** the flow is assumed to be inviscid. This means no viscous effects are taken into account.
- **Incompressible Flow:** the flow is assumed to be incompressible. This means the density throughout the streamtube is constant. This enables the use of Bernoulli's equation in locations of a continuous pressure distribution. This also results in the product of area and flow velocity being constant over the flow.
- **2D Flow:** it is assumed that the flow characteristics can accurately be modeled using 2 dimensional flow characteristics.
- **Constant Internal Energy:** t is assumed that the internal energy within the streamtube is constant so there no radiation, convection or conduction occurring.
- **Independent annulus:** the annuli are considered independently of one another. In reality, flow characteristics on one annulus will influence the characteristics on another (cross flow), this effect is ignored.
- **Circular Discs:** the actuator disc in the model is assumed to be of circular shape. In reality slight changes in the shape could occur, these are neglected.

Results 4

4.1 Reference Data (for cl(alpha) and cd(alpha))

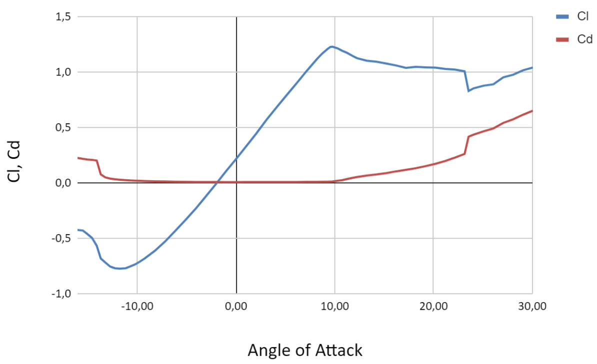

Figure 4.1: Caption

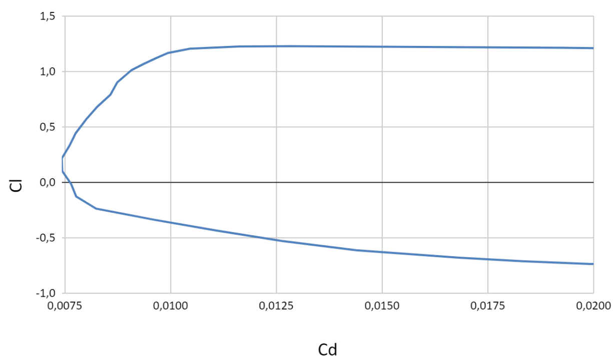

Figure 4.2: Caption

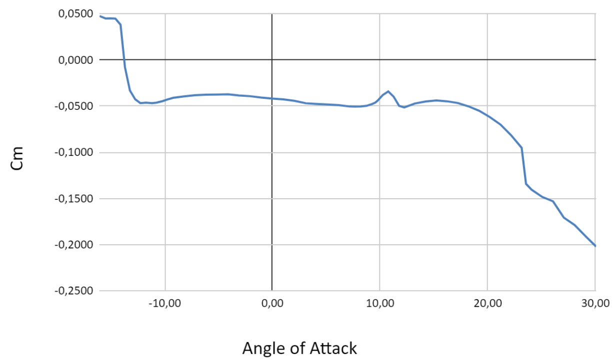

Figure 4.3: Caption

Conclusion 5

6 Python Script