

### 3-nonlinear-dynamics-stability

goal: develop qualitative & quantitative tools to study nonlinear dynamics

topics:

1°. nonlinear dynamics

[Nv7 ch 2]

[AMv2 ch 5]

1<sup>1</sup>. trajectories & visualization

1<sup>2</sup>. equilibrium trajectories

2°. stability

2<sup>1</sup>. definition of stability

[Nv7 ch 6]

2<sup>2</sup>. stability of linear DE

[AMv2 ch 6]

2<sup>3</sup>. parametric stability

[Nv7 ch 8]

\*read [AMv2 ch 5.4] to learn about an advanced test for stability

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1°. nonlinear control systems

◦ consider a DE with state  $x \in \mathbb{R}^n$ , input  $u \in \mathbb{R}^p$

$$\dot{x} = f(x, u)$$

◦ the simplest type of feedback sets  $u$  as a function of  $x$ , e.g.  $u = \alpha(x)$ , so

$$\dot{x} = f(x, \alpha(x)) = F(x)$$

→ we'll focus on  $\dot{x} = F(x)$  below

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1<sup>1</sup>. trajectories & visualization

–  $x: [0, \infty) \rightarrow \mathbb{R}^n$  is a trajectory of DE with initial state  $x(0)$

1. trajectories & visualization

-  $x: [0, \infty) \rightarrow \mathbb{R}^n$  is a trajectory of DE with initial state  $x(0)$

if  $\forall t \geq 0: \dot{x}(t) = F(x(t))$   
↳ so  $x$  is continuous and differentiable

\* function  $x$  satisfies an infinite number of equations

↳ we'll assume unique trajectories always exist,  
but this isn't guaranteed in general; see [AMv2 ex 5.2, 5.3]

ex: damped oscillator (i.e. RLC circuit; spring-mass-damper)

$$\ddot{q} + 2\zeta\omega_0 \dot{q} + \omega_0^2 q = 0$$

- assume  $\zeta < 1$ , i.e. lightly damped

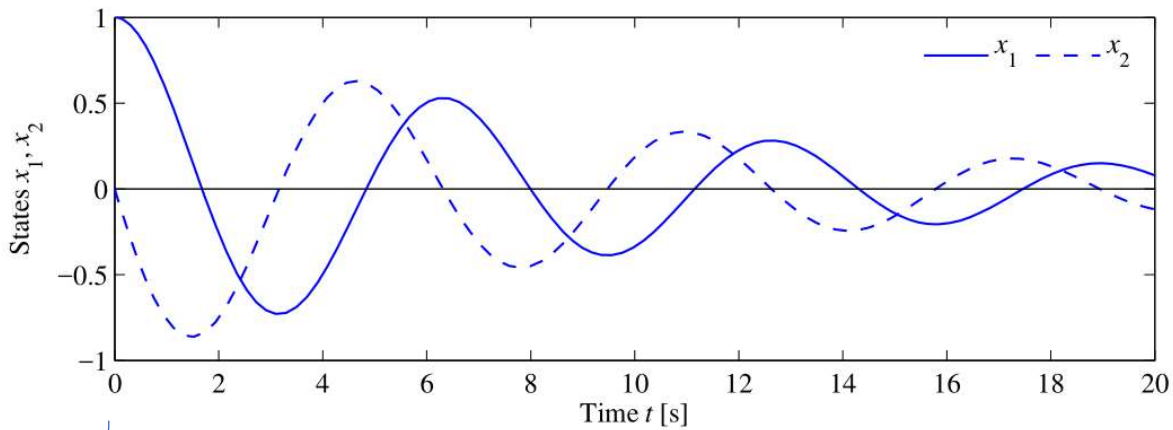
- setting  $x = (x_1, x_2) = (q, \dot{q}/\omega_0)$ ,

$$\dot{x} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} \omega_0 x_2 \\ -\omega_0 x_1 - 2\zeta\omega_0 x_2 \end{bmatrix} = F(x)$$

$$\Rightarrow x(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix}$$

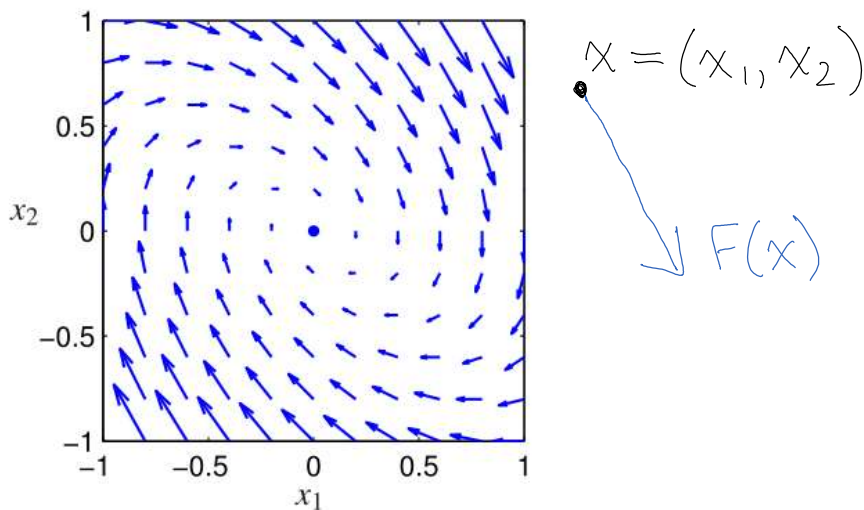
$$= e^{-\zeta\omega_0 t} \begin{bmatrix} x_1(0) \cos \omega_d t + \frac{1}{\omega_d} (\omega_0 \zeta x_1(0) + x_2(0)) \sin \omega_d t \\ x_2(0) \cos \omega_d t + \frac{1}{\omega_d} (\omega_0^2 x_1(0) + \omega_0 \zeta x_2(0)) \sin \omega_d t \end{bmatrix}$$

→ verify this function satisfies DE for all  $t \geq 0$



↳ initial state  $x(0) = (x_1(0), x_2(0)) = (1, 0)$

- since  $x \in \mathbb{R}^2$ ,  $\dot{x} = F(x)$  can be visualized as a vector field

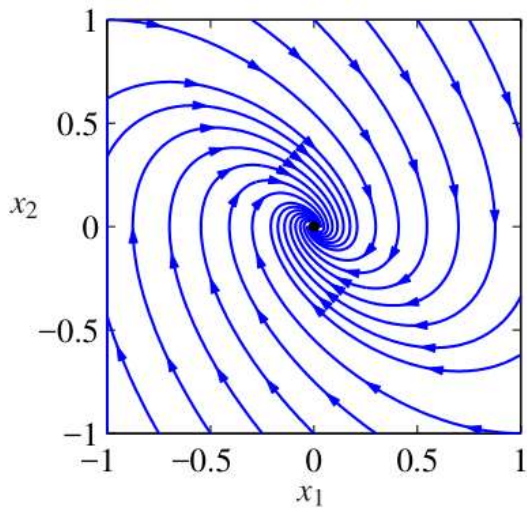


- $F(x)$  specifies the direction (& rate) of change in  $x$ :



- overlaying a representative collection of trajectories yields a phase portrait

↳ "phase" terminology is inherited from physics (classical mechanics)



1.2 equilibrium trajectories

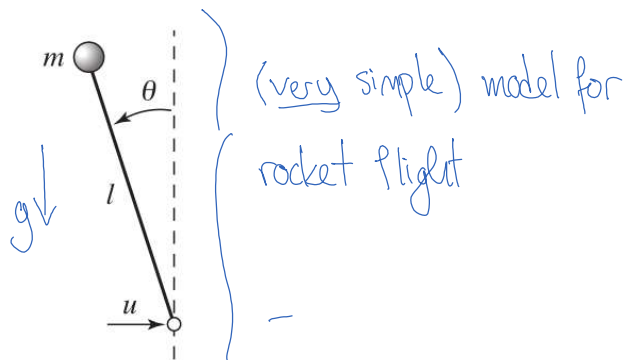
• some trajectories don't move — we say they are at equilibrium:

$$\dot{x}_e = F(x_e) = 0$$

→ importantly, if system is initialized at  $x_e$ , it stays there:

$$x(0) = x_e \Rightarrow x(t) = x_e, \text{ all } t \geq 0$$

ex: inverted pendulum



— state  $x = (\theta, \dot{\theta})$

— input  $u$  — horizontal acceleration

of pivot point

$$- ml^2 \ddot{\theta} = mgl \sin \theta - \gamma \dot{\theta} + l u \cos \theta$$

where  $\gamma$  is coefficient of rotational friction

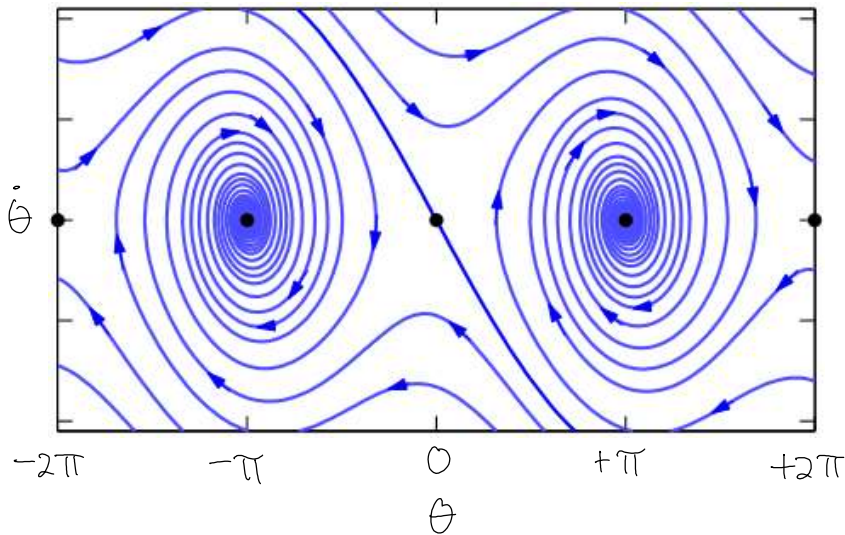
- with no input ( $u=0$ ),

$$\dot{x}_e = (\dot{\theta}_e, \ddot{\theta}_e) = 0 \Leftrightarrow \dot{\theta}_e = 0, \sin \theta_e = 0,$$

$$\text{so } \theta_e = n\pi, \quad n \in \{0, +1, -1, +2, -2, \dots\} = \mathbb{Z}$$

→ what is the physical configuration  
when  $n$  even?  $n$  odd?

- phase portrait:



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2°: stability

- a fundamental goal of feedback is to ensure stability, that is, to ensure trajectories converge to steady-state

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2': definition of stability

- given an equilibrium  $x_e$  s.t.  $\dot{x}_e = F(x_e) = 0$ ,

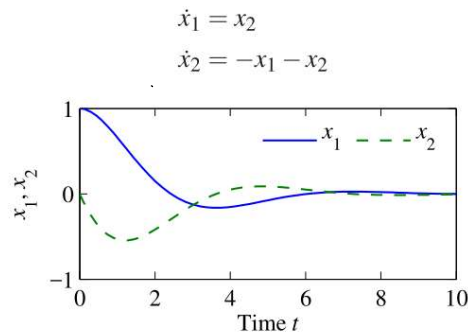
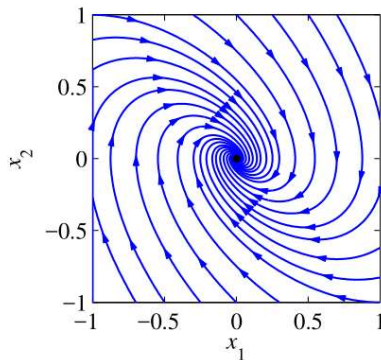
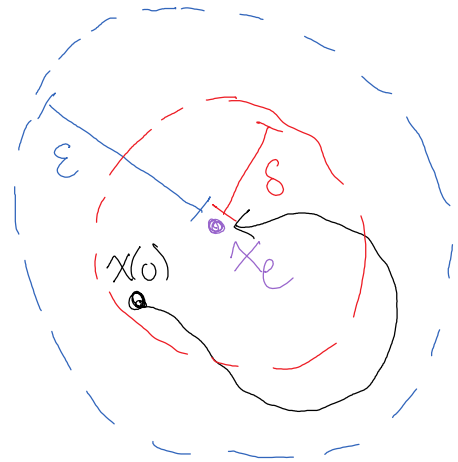
we'll say that  $x_e$  is stable if trajectories that start close stay close & get closer over time:

$\forall \varepsilon > 0$  :  $\leftarrow$  closeness of  $x(t)$  to  $x_e$

$\exists \delta > 0$  :  $\leftarrow$  closeness of  $x(0)$  to  $x_e$

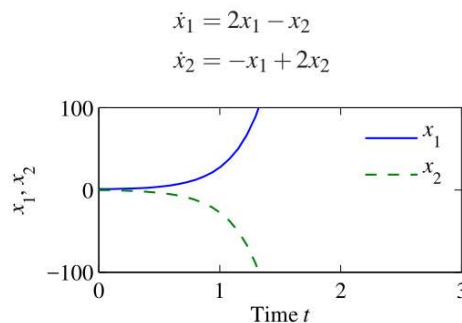
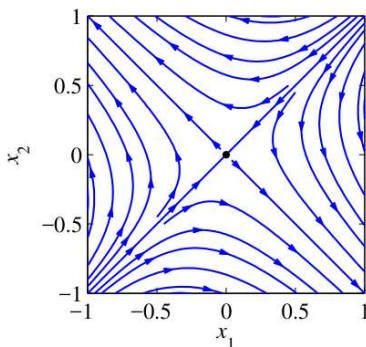
$$\|x(0) - x_e\| < \delta \Rightarrow \|x(t) - x_e\| < \varepsilon$$

and  $x(t) \rightarrow x_e$  as  $t \rightarrow \infty$



- if  $\delta$  cannot be made arbitrarily large (when  $\varepsilon$  is large), stability is local; otherwise, stability is global

• if a trajectory is not stable, then we term it unstable:



## 2<sup>2</sup>. stability of linear DE

◦ consider  $\dot{x} = Ax$

→ what is the shape of  $A$ ?

– note that  $0$  (zero; the origin) is an equilibrium for every matrix  $A$ , since

$$\dot{0} = A \cdot 0 = 0$$

→ are there other equilibria  $x_e \neq 0$ ?

(what does  $Ax_e = 0$  tell you about the matrix  $A$ ?)

– it turns out that stability of the origin is determined by the set of eigenvalues of  $A$ :

$$\lambda(A) = \{s \in \mathbb{C} \mid \exists v \in \mathbb{C}^n : Av = sv\}$$

↳  $v$  is an eigenvector  
w/ eigenvalue  $s$

$$= \{s \in \mathbb{C} \mid \det(sI - A) = 0\}$$

↳  $\det: \mathbb{C}^{n \times n} \rightarrow \mathbb{R}$  is the determinant function

– recall that  $\det(sI - A)$  is a polynomial expression in  $s$ ,

$$\det(sI - A) = a_0 s^n + a_1 s^{n-1} + \dots + a_{n-1} s + a_n,$$

termed the characteristic polynomial ← this terminology is not

$$p(s) = u_0 + u_1 s + \dots + u_{n-1} s^{n-1} + u_n s^n$$

termed the characteristic polynomial

← this terminology is not accidental — related to characteristic polynomial of transfer functions

- since  $\lambda(A)$  is the set of roots of the  $n$ -th degree polynomial  $\det(sI - A)$ , there are (at most)  $n$  distinct eigenvalues:

$$\lambda(A) = \{ \lambda_1, \dots, \lambda_j, \dots, \lambda_n \}$$

→ if  $s \in \lambda(A)$  and  $A \in \mathbb{R}^{n \times n}$ ,

show that  $s^* \in \lambda(A)$

( $s^*$  denotes complex-conjugate of  $s \in \mathbb{C}$ )

- to see how eigenvalues determine stability, consider the special case of diagonal  $A$ :

$$\dot{x} = Ax = \begin{bmatrix} \lambda_1 & 0 \\ 0 & \lambda_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

— clearly  $\dot{x}_1 = \lambda_1 x_1$  ← doesn't depend on  $x_2$

$\dot{x}_2 = \lambda_2 x_2$  ← doesn't depend on  $x_1$

$$\text{so } x_1(t) = e^{\lambda_1 t} x_1(0), \quad x_2(t) = e^{\lambda_2 t} x_2(0)$$

→ what condition on  $\lambda_1, \lambda_2$  ensure  $x = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$  is stable?

— more generally for  $x \in \mathbb{R}^n$ ,  $A \in \mathbb{R}^{n \times n}$ :

$$\dot{x} = Ax = \begin{bmatrix} \lambda_1 & 0 & \dots & 0 \\ 0 & \lambda_2 & \dots & 0 \\ 0 & 0 & \lambda_3 & \dots & 0 \\ \vdots & \vdots & \vdots & \ddots & \vdots \\ 0 & \dots & \lambda_{n-1} & 0 \\ 0 & \dots & 0 & \lambda_n \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ \vdots \\ x_{n-1} \\ x_n \end{bmatrix}$$



$$\begin{bmatrix} \dot{x}_1 & \dots & \dot{x}_{n-1} \\ 0 & \dots & 0 \end{bmatrix} = \begin{bmatrix} \lambda_1 & \dots & \lambda_{n-1} \\ 0 & \dots & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ \dots \\ x_{n-1} \end{bmatrix}$$

- clearly  $\dot{x}_j = \lambda_j x_j$  (independent of  $x_{i \neq j}$ )

so  $x_j(t) = e^{\lambda_j t} x_j(0)$ ,

so origin is stable if  $\lambda_j < 0$ , all  $j \in \{1, \dots, n\}$

• another special case:

-  $\dot{x} = \begin{bmatrix} \sigma & \omega \\ -\omega & \sigma \end{bmatrix} x = Ax$

→ determine eigenvalues of  $A$   
(recall  $\det \begin{bmatrix} a & b \\ c & d \end{bmatrix} = ad - bc$ )

- we conclude that

$$x_1(t) = e^{\sigma t} (x_1(0) \cos \omega t + x_2(0) \sin \omega t)$$

$$x_2(t) = e^{\sigma t} (-x_1(0) \sin \omega t + x_2(0) \cos \omega t)$$

→ what condition on  $\sigma$  &  $\omega$  ensure the origin is stable?

- more generally,  $A$  could be block-diagonal:

$$\dot{x} = Ax = \begin{bmatrix} \sigma_1 & \omega_1 & & 0 \\ -\omega_1 & \sigma_1 & & 0 \\ & & \ddots & \\ 0 & & & \sigma_m & \omega_m \\ & & & -\omega_m & \sigma_m \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ \vdots \\ x_{2m-1} \\ x_{2m} \end{bmatrix}$$

- now eigenvalues  $\lambda(A) = \{ \sigma_k \pm j\omega_k \}_{k=1}^m$  and

$$x_{2k-1}(t) = e^{\sigma_k t} (x_{2k-1}(0) \cos \omega_k t + x_{2k}(0) \sin \omega_k t)$$

$$x_{2k}(t) = e^{\sigma_k t} (-x_{2k-1}(0) \sin \omega_k t + x_{2k}(0) \cos \omega_k t)$$

so origin is stable if  $\sigma_k = \operatorname{Re} \lambda_k$

• most systems aren't (block) diagonal, but many can be transformed to be:

– if  $\lambda(A)$  consists of  $n$  distinct eigvals, there exists invertible  $T \in \mathbb{R}^{n \times n}$   
 such that  $TAT^{-1} \in \mathbb{R}^{n \times n}$  is (block) diagonal i.e.  $\det T \neq 0$

– applying the change-of-coordinates  $z = Tx$   
 yields  $\dot{z} = T\dot{x} = TAx = TAT^{-1}z$

→ show that  $\lambda(A) = \lambda(TAT^{-1})$

– now given a trajectory  $z: [0, \infty) \rightarrow \mathbb{R}^n$  for  $\dot{z} = TAT^{-1}z$ ,  
 $x: [0, \infty) \rightarrow \mathbb{R}^n$  defined by  $x(t) = T^{-1}z(t)$   
 is a trajectory for  $\dot{x} = Ax$ ,  
 so stability of  $z$  determines stability of  $x$

### 2.3. parametric stability

• when designing a feedback controller, we've seen that model parameters  
 can limit stabilizing controller parameters

(e.g. PI control of 2nd-order system)

– consider  $\dot{x} = F(x; \mu)$  where  $x \in \mathbb{R}^n$  – states semicolon implies  $\mu$  doesn't vary with time

$\mu \in \mathbb{R}^k$  - parameters  
 \* vector  $\mu$  can have model parameters (e.g.  $R, L, C; M, C, K; \dots$ )  
 and/or controller parameters (e.g.  $k_p, k_I, k_D, \dots$ )

- since equilibrium  $x_e$  satisfies  $\dot{x}_e = F(x_e; \mu) = 0$ ,  
 the equilibrium generally varies with parameters:  $x_e(\mu)$

• in linear systems, the equilibrium won't move:  $\dot{0} = A(\mu) \cdot 0 = 0$   
 (we're always interested in  $x_e = 0$ )

- plotting eigenvalues  $\lambda(A(\mu)): \mathbb{R} \Rightarrow \mathbb{C}$  special notation for multi-valued function  
 as parameter  $\mu$  varies:  $\mu \mapsto \{\lambda_j\}_{j=1}^n$

is termed a root locus diagram — root = eigenvalues  
locus = image/graph of eigenvalues in  $\mathbb{C}$

ex: proportional-integral control of first-order system

•  $P(s) = \frac{b}{s+a}$      •  $C(s) = k_p + k_I \frac{1}{s}$

•  $G_{yu} = \frac{P}{1+PC} = \frac{bs}{s^2 + (a+bk_p)s + bk_I}$

• choosing parameters  $a=1, b=1, k_p=10$ ,  
 roots of  $s^2 + (a+bk_p)s + bk_I$

system is stable  
 when all roots have  
 negative real part

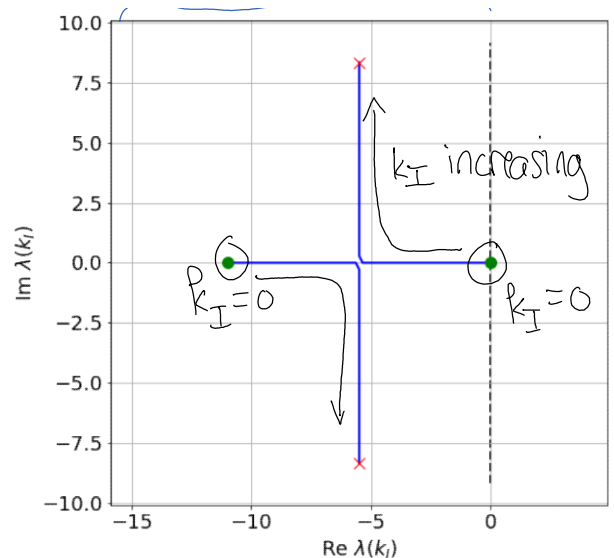


roots of  $s^2 + (a + bk_p)s + bk_I$

are  $-\frac{11}{2} \pm \frac{1}{2} \sqrt{121 - 4k_I}$ ;

plotting these in complex plane as a function of  $k_I$  yields:

"root locus" diagram  $\longrightarrow$



\* indicates that  $k_I$  can be arbitrarily large, but that isn't physically realistic — large  $k_I$  will yield large inputs, which can excite unmodeled dynamics (of sensors, actuators, E&M coupling, vibrations, ...)

including unmodeled dynamics yields  $P(s) = \frac{b}{(sta)(1+st)}$

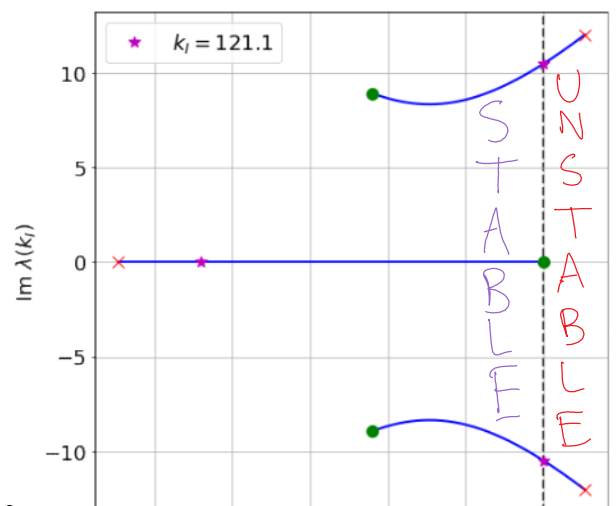
$$\text{so } G_{yv} = \frac{P}{1+PC} = \frac{bs}{Ts^3 + (1+aT)s^2 + (a+bk_p)s + bk_I}$$

using parameters  $a=1, b=1, T=\frac{1}{10}, k_p=10$

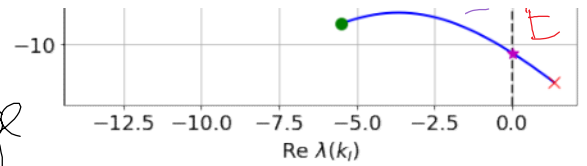
and plotting roots of  $G_{yv}$  in complex plane yields:

\* importantly, two roots leave the left half-plane

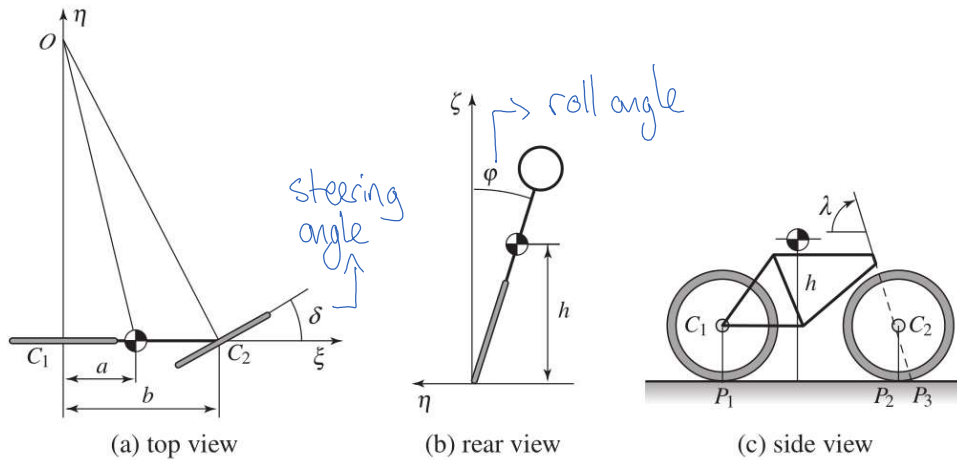
for  $k_I > 121.1$ , so the system



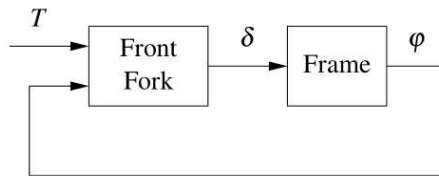
for  $K_I > 121.1$ , so the system is unstable when  $K_I$  is too large



ex: (linearized) bicycle



**Figure 4.5:** Schematic views of a bicycle. The steering angle is  $\delta$ , and the roll angle is  $\varphi$ . The center of mass has height  $h$  and distance  $a$  from a vertical through the contact point  $P_1$  of the rear wheel. The wheel base  $b$  is the distance between  $P_1$  and  $P_2$ , and the trail  $c$  is the distance between  $P_2$  and  $P_3$ .



**Figure 4.6:** Block diagram of a bicycle with a front fork. The steering torque applied to the handlebars is  $T$ , the roll angle is  $\varphi$  and the steering angle is  $\delta$ . Notice that the front fork creates a feedback from the roll angle  $\varphi$  to the steering angle  $\delta$  that under certain conditions can stabilize the system.

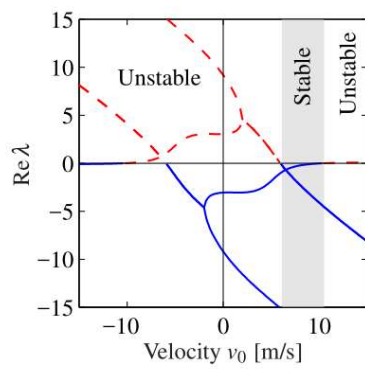
- whipple (linearized) model velocity of rear wheel

$$M \begin{bmatrix} \ddot{\varphi} \\ \ddot{\delta} \end{bmatrix} + C v_0 \begin{bmatrix} \dot{\varphi} \\ \dot{\delta} \end{bmatrix} + (K_0 + K_2 v_0^2) \begin{bmatrix} \varphi \\ \delta \end{bmatrix} = \begin{bmatrix} 0 \\ T \end{bmatrix}$$

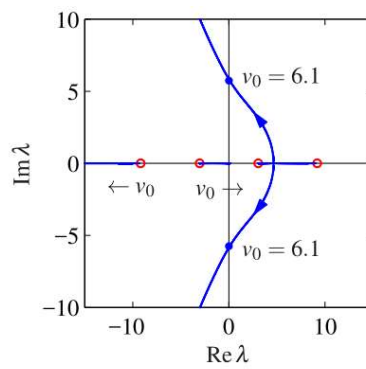
$\in \mathbb{R}^{2 \times 2}$        $\in \mathbb{R}^{2 \times 2}$        $\in \mathbb{R}^{2 \times 2}$

-  $x = (x_1, x_2, x_3, x_4) = (\varphi, \delta, \dot{\varphi}, \dot{\delta})$  identity matrix

$$\Rightarrow \dot{x} = A x = \begin{bmatrix} 0 & I \\ -M^{-1}(K_0 + K_2 v_0^2) & -M^{-1} C v_0 \end{bmatrix} x$$



(a) Stability diagram



(b) Root locus diagram

**Figure 5.18:** Stability plots for a bicycle moving at constant velocity. The plot in (a) shows the real part of the system eigenvalues as a function of the bicycle velocity  $v_0$ . The system is stable when all eigenvalues have negative real part (shaded region). The plot in (b) shows the locus of eigenvalues on the complex plane as the velocity  $v$  is varied and gives a different view of the stability of the system. This type of plot is called a *root locus diagram*.