

_7-transfer-functions

goal: frequency-domain tools and concepts for analysis & control

1°. frequency-domain modeling

1°. transfer function of an LTI system [AMv2 Ch 6.3, 9.2] [Nv7 Ch 4.1]

1°. block diagrams [AMv2 Ch 9.4] [Nv7 Ch 5]

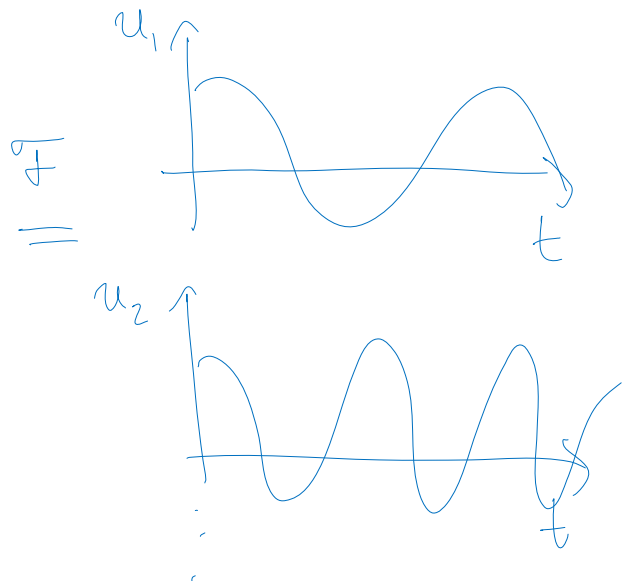
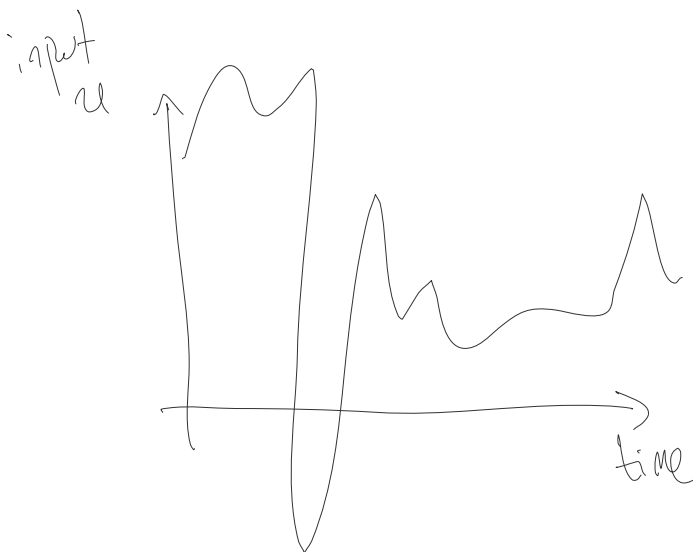
1°. poles & zeros [AMv2 Ch 9.5] [Nv7 Ch 4.2, 4.10]

1°. Bode plot [AMv2 Ch 9.6] [Nv7 Ch 10.1]

1°. frequency domain modeling

• key idea: assuming system is stable, represent/analyze its steady-state response to periodic (sinusoidal) input

* LTI systems' response to complex input is obtained by superimposing response to pure sinusoids



1°. transfer function of LTI system

1. transfer function of LTI system

• consider the response of LTI system to input signal $u(t)$

$$\dot{x} = Ax + Bu, \quad y = Cx + Du$$

fact: $x(t) = e^{At}x(0) + \int_0^t e^{A(t-\tau)} Bu(\tau) d\tau$ } convolution equation

intuition: input $u(\tau)$ applied at time τ adds system's response $e^{A(t-\tau)} Bu(\tau)$ to "initial state" $\hat{x}(\tau) = Bu(\tau)$

→ verify by differentiating w.r.t. time t

when input $u \equiv 0$, $x(t) = e^{At}x(0)$

if $A = T\Lambda T^{-1}$ where $\Lambda = \text{diag}(\lambda_1, \lambda_2, \dots, \lambda_n)$

then $e^{At} = T e^{\Lambda t} T^{-1}$, $e^{\Lambda t} = \text{diag}(e^{\lambda_1 t}, e^{\lambda_2 t}, \dots, e^{\lambda_n t})$

thus: $y(t) = Cx(t) + Du(t)$ note: $u \in \mathbb{R}^p$, $y \in \mathbb{R}^q$

* linear in $x(0)$ $\frac{1}{2} u$ $\left\{ \begin{array}{l} = \underbrace{C e^{At} x(0)}_{\text{homogeneous response to } x(0)} + \underbrace{\int_0^t C e^{A(t-\tau)} B u(\tau) d\tau + Du(t)}_{\text{particular response to } u} \end{array} \right.$

$u_0 \in \mathbb{R}^p$ to $u \in \mathbb{R}^p$

• now we want $G_{yu}(s)$ s.t. $u(t) = u_0 e^{st} \leadsto y(t) = e^{st} \cdot G_{yu}(s) \cdot u_0$

* recall $\cos(\omega t) = \frac{1}{2}(e^{j\omega t} + e^{-j\omega t})$, $\sin(\omega t) = \frac{1}{2j}(e^{j\omega t} - e^{-j\omega t})$

$$\begin{aligned} y(t) &= C e^{At} x(0) + \int_0^t C e^{A(t-\tau)} B u_0 e^{s\tau} d\tau + D u_0 e^{st} \\ &= C e^{At} x(0) + C e^{At} \left[\int_0^t \underbrace{e^{-A\tau} e^{s\tau}}_{e^{(sI-A)\tau}} d\tau \right] B u_0 + D u_0 e^{st} \end{aligned}$$

→ evaluate the integral — hint: find expression that has $e^{(sI-A)\tau}$ as its derivative

[a simpler problem: consider $A = \Lambda = \text{diag}(\lambda_1, \lambda_2, \dots, \lambda_n)$]

• know: $\frac{d}{d\tau} e^{(sI-A)\tau} = (sI-A) e^{(sI-A)\tau}$
 so $\frac{d}{d\tau} [(sI-A)^{-1} e^{(sI-A)\tau}] = e^{(sI-A)\tau}$ $\left\{ \begin{array}{l} x \cdot e^x = e^x \cdot x \\ e^x x^{-1} = x^{-1} e^x \end{array} \right.$

* $s \notin \{ \lambda \in \mathbb{C} : \det(sI-A) = 0 \}$

$$\Rightarrow \int_0^t e^{(sI-A)\tau} d\tau = (sI-A)^{-1} e^{(sI-A)\tau} \Big|_{\tau=0}^{\tau=t} \\ = (sI-A)^{-1} (e^{(sI-A)t} - I)$$

$\underbrace{e^{(sI-A)t}}_{e^{-At} \cdot e^{st}}$

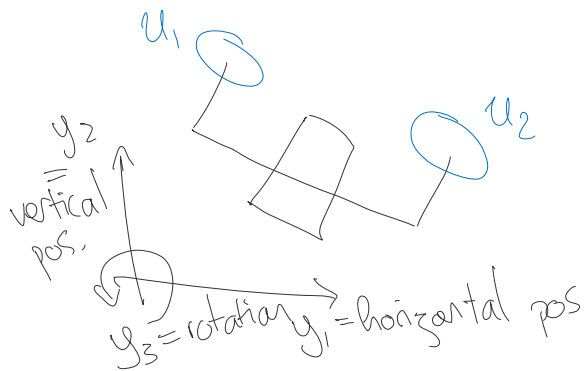
$$\Rightarrow y(t) = \underbrace{C e^{At} (x(0) - (sI-A)^{-1} B)}_{\text{transient response} \rightarrow 0 \text{ if } A \text{ stable}} + \underbrace{[C (sI-A)^{-1} B + D] u_0 e^{st}}_{\text{steady-state response}} \\ G_{yu}(s) \in \mathbb{R}^{p \times p}$$

* if A is stable, know $e^{At} \rightarrow 0$ as $t \rightarrow \infty$
 so $y(t) \rightarrow G_{yu}(s) u_0 e^{st}$ as $t \rightarrow \infty$

note: $G_{yu}(s) \in \mathbb{R}^{p \times p}$; the transfer function from input u_j to output y_i is $[G_{yu}(s)]_{ij}$

* $G_{yu}(s) = C(sI-A)^{-1}B + D$ gives a recipe for determining transfer function matrix from matrices A, B, C, D

ex: quadrotor



$$u = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}, \quad y = \begin{bmatrix} y_1 \\ y_2 \\ y_3 \end{bmatrix}$$

$$G_{yu} = \begin{bmatrix} G_{y_1 u_1} & G_{y_1 u_2} \\ G_{y_2 u_1} & G_{y_2 u_2} \\ G_{y_3 u_1} & G_{y_3 u_2} \end{bmatrix} \in \mathbb{R}^{3 \times 2}$$

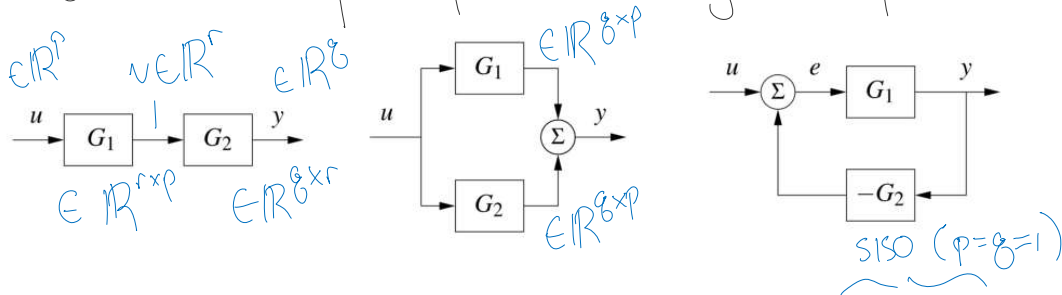
* applying
gives

$$u_1 = e^{s_1 t}, \quad u_2 = e^{s_2 t}$$

$$y_3(t) = \theta(t) = G_{y_3 u_1}(s_1) \cdot e^{s_1 t} + G_{y_3 u_2}(s_2) \cdot e^{s_2 t}$$

12. block diagrams

• we've seen examples of block diagram representation & algebra



$$(a) G_{yu}(s) = G_2(s)G_1(s)$$

$$(b) G_{yu}(s) = G_1(s) + G_2(s)$$

$$(c) G_{yu}(s) = \frac{G_1(s)}{1 + G_1(s)G_2(s)}$$

$$= (I + G_2 G_1)^{-1} G_1$$

→ rederive / verify these formulas for $p, g \neq 1$

$$(c) y = G_1 e = G_1 (u - G_2 y) = G_1 u - G_1 G_2 y$$

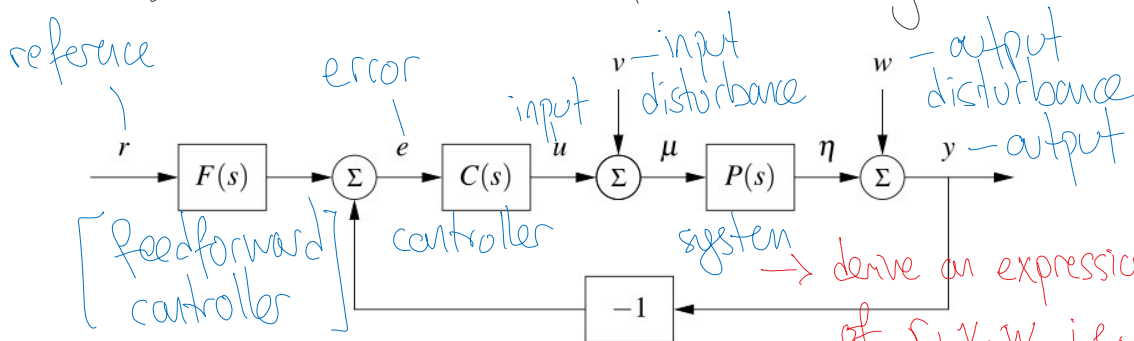
$$\Leftrightarrow y + G_1 G_2 y = G_1 u$$

$$= (I + G_1 G_2) y = G_1 u \Leftrightarrow$$

$$\Leftrightarrow y = (I + G_1 G_2)^{-1} G_1 u \quad \text{so } G_{yu} = (I + G_1 G_2)^{-1} G_1$$

$$y \in \mathbb{R}^{1 \times g} \quad y + y H_1 H_2 = y (I + H_1 H_2)$$

• considers a more detailed feedback diagram



→ derive an expression for e in terms of r, v, w , i.e. find G_{er}, G_{ev}, G_{ew} s.t. $e = G_{er} r + G_{ev} v + G_{ew} w$

ex: cruise control

r : desired speed

u : control input, i.e. torque / gas pedal

C : cruise control

P : engine

u : control input, ie torque / gas pedal
 v : input disturbance, eg headwind, slope
 w : output disturbance, eg sensor noise, ADC quantization
 y : output (speed)

P : car dynamics
 torque \rightarrow speed
 eg $P = \frac{b}{s+a}$

$$\begin{aligned}
 e &= Fr - y = Fr - (w + y) = Fr - w - Py \\
 &= Fr - w - P(v + u) = Fr - w - Pv - PCe
 \end{aligned}$$

$$\Leftrightarrow e + PCe = Fr - w - Pv$$

$$\Leftrightarrow (I + PC)e = Fr - w - Pv$$

$$\Leftrightarrow e = (I + PC)^{-1} (Fr - w - Pv)$$

$$\begin{aligned}
 y \neq 1 &= \underbrace{(I + PC)^{-1} F}_{{G_{er}}} r - \underbrace{(I + PC)^{-1} w}_{{G_{ew}}} - \underbrace{(I + PC)^{-1} P}_{{G_{ev}}} v \\
 &= \boxed{\frac{G_{er} F}{1 + PC}} r - \boxed{\frac{1}{1 + PC}} w - \boxed{\frac{P}{1 + PC}} v
 \end{aligned}$$

$$y = \frac{(w + Pv + PCFr)}{1 + PC} = \underbrace{\frac{1}{1 + PC}}_{{G_{yw}}} w + \underbrace{\frac{P}{1 + PC}}_{{G_{yv}}} v + \underbrace{\frac{PCF}{1 + PC}}_{{G_{yr}}} r$$

not a transfer function
 b/c it mixes signals (w, v, r)
 with transforms

1.3 poles & zeros

consider the SISO transfer function $G(s) = \frac{b(s)}{a(s)}$

G is a rational function that is ratio of polynomials

- consider the transfer function $G(s) = \frac{a(s)}{b(s)}$
- * G is a rational function, that is, ratio of polynomials
- roots of a are termed poles
- roots of b are termed zeros
- difference between degree of a & b termed relative degree
- G is proper if ≥ 0 , strictly proper if > 0

• let's relate these concepts to $\dot{x} = Ax + Bu$, $y = Cx + Du$

- poles = $\lambda(A)$, i.e. eigenvalues of A

- zeros are complex numbers $s \in \mathbb{C}$ s.t.

$$u(t) = e^{st} \text{ yields } y(t) = 0$$

* if state is fully actuated (B square)
or fully measured (C square)

then there are no zeros

• since poles & zeros govern system's behavior,
often visualize them with a pole-zero diagram:

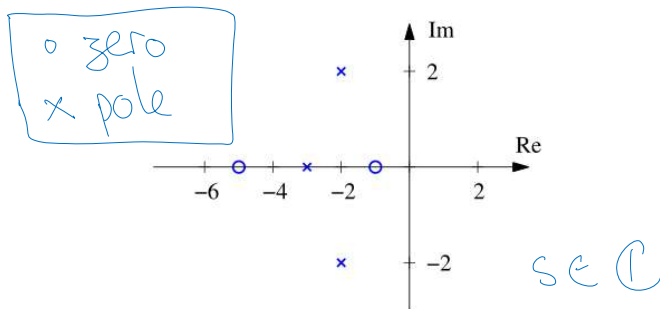


Figure 9.9: A pole zero diagram for a transfer function with zeros at -5 and -1 and poles at -3 and $-2 \pm 2j$. The circles represent the locations of the zeros, and the crosses the locations of the poles. A complete characterization requires we also specify the gain of the system.

14. Bode plot

• since a complex exponential input $e^{j\omega t}$ yields

1. Bode plot

• since a complex exponential input $e^{j\omega t}$ yields
output $G(j\omega)e^{j\omega t}$
 $= |G(j\omega)|e^{j\omega(t + \angle G(j\omega))}$

it's useful to visualize
gain $|G(j\omega)|$ and phase $\angle G(j\omega)$
as frequency ω varies

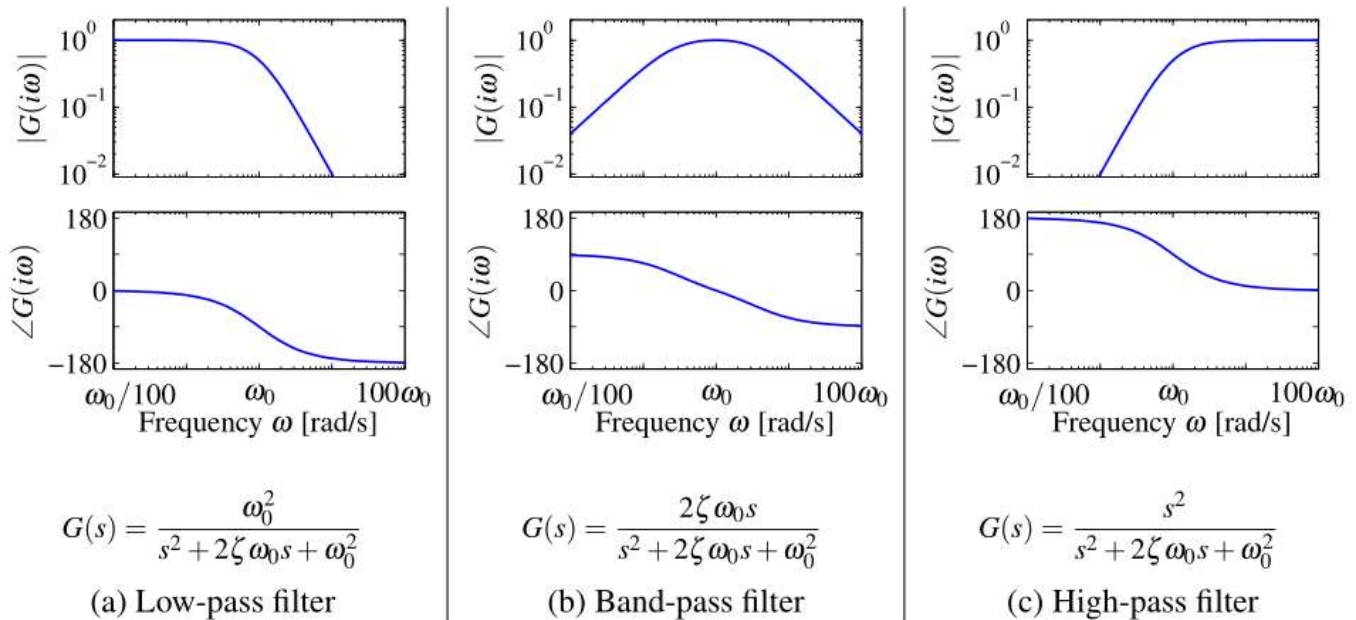


Figure 9.17: Bode plots for low-pass, band-pass, and high-pass filters. The upper plots are the gain curves and the lower plots are the phase curves. Each system passes frequencies in a different range and attenuates frequencies outside of that range.

→ which of these would you want for G_{yr} , G_{yv} , G_{yw} ?
(in cruise control example)