

## 10-proportional-integral-derivative

goal: techniques for analysis, design,  
& implementation of the most  
ubiquitous control architecture

*Based on a survey of over eleven thousand controllers in the refining, chemicals and pulp and paper industries, 97% of regulatory controllers utilize a PID feedback control algorithm.*

L. Desborough and R. Miller, 2002 [DM02a].

1°. essentials of feedback control

[AMU2 ch 11] [Nu7 ch 9.4]

1<sup>1</sup>. a simple controller

1<sup>2</sup>. implementation issues

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1°. essentials of feedback control

- let's take a step back and reflect on (a) what feedback controllers do and (b) how they do it

(a) starting in the first week of lecture & extending to last week, we've studied how feedback can be used to:

- make output ( $y$ ) track reference ( $r$ )
- reject disturbances to

- inputs ( $v$ ) and outputs ( $w$ )
- provide robustness to model inaccuracies, e.g. unmodeled dynamics or uncertain parameters

(b) feedback controllers can use

- prior knowledge, e.g. of process dynamics, disturbance statistics, or model uncertainty
- the measured signal  $y$  and commanded signals  $u, r$ , including their time histories and (predictions about their) futures

ex: the full-state observer/controller processes the time history of the output and predicts the future:

$$\dot{\hat{x}} = A\hat{x} + Bu + L(y - \hat{y})$$

$$\hat{y} = C\hat{x} + Du$$

- requires substantial prior knowledge (i.e.  $A, B, C, D$ ), and is only reliable if uncertainty is low

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1'. a simple controller

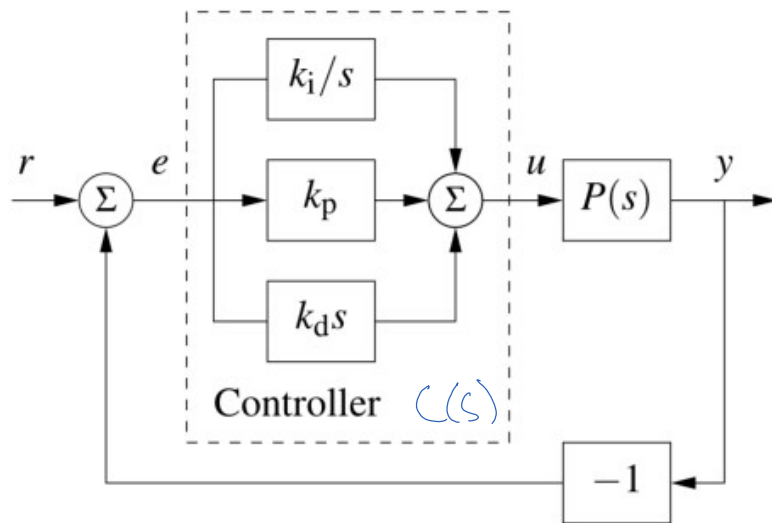
- at another extreme of complexity, consider the simplest controller,

obtained using a linear combination

of: P – present error

I – (integral of) past error

D – (prediction of) future error  
(using linear extrapolation)

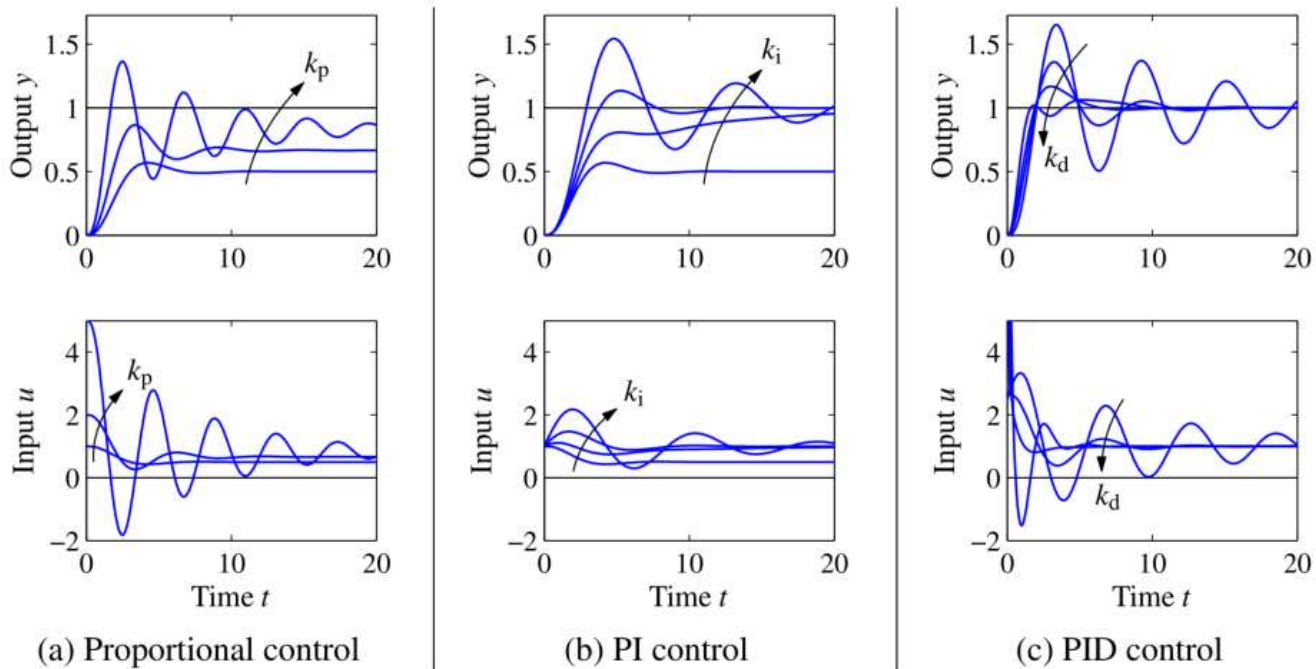


– as a transfer function:

$$C(s) = k_p + \frac{k_I}{s} + k_D s$$

– as a function of time:

$$u(t) = k_p e(t) + k_I \int_0^t e(\tau) d\tau + k_D \frac{d}{dt} e(t)$$



**Figure 11.2:** Responses to step changes in the reference value for a system with a proportional controller (a), PI controller (b) and PID controller (c). The process has the transfer function  $P(s) = 1/(s+1)^3$ , the proportional controller has parameters  $k_p = 1, 2$  and  $5$ , the PI controller has parameters  $k_p = 1, k_i = 0.2, 0.5$ , and  $1$ , and the PID controller has parameters  $k_p = 2.5, k_i = 1.5$  and  $k_d = 0, 1, 2$ , and  $4$ .

→ describe trends in transient & steady-state error as  $k_p, k_i, k_d$  vary

- integral feedback eliminates steady-state error (assuming stable closed-loop system)

claim: if  $\lim_{t \rightarrow \infty} u(t) = \bar{u}$ ,  $\lim_{t \rightarrow \infty} e(t) = \bar{e}$ , then  $\bar{e} = 0$

→ prove this claim using the expression for  $u(t)$

→ prove this claim using the transfer function  $C(s)$

- derivative feedback predicts future error
  - proportional + derivative terms are

$$\left. \begin{aligned} k_p e(t) + k_D \frac{d}{dt} e(t) &= k_p (e(t) + \Delta \dot{e}(t)) \\ &\approx k_p e(t + \Delta) \end{aligned} \right\} \Delta = \frac{k_D}{k_p}$$

Q: why is PID ubiquitous?

A: it's (often) good enough

\* if there are  $\leq 2$  unstable poles,  
(e.g. if the system is already stable)  
then PID is a simple, intuitive way to  
shape closed-loop performance

12. implementation issues

\* we'll sample a few issues - read  
[AMv2 Ch 11.5] for more!

• we have an elegant formula for

$$u(t) = k_p e(t) \quad \text{proportional}$$

$$+ k_I \int_0^t e(\tau) d\tau \quad \text{integral}$$

$$+ k_D \frac{d}{dt} e(t) \quad \text{derivative}$$

control, with nice analytical properties

→ what practical issues could  
arise when this formula is  
implemented on actual hardware,  
e.g. digital microcontroller;

## analogue RLC/op-amp circuit?

(i) integral term is unbounded,

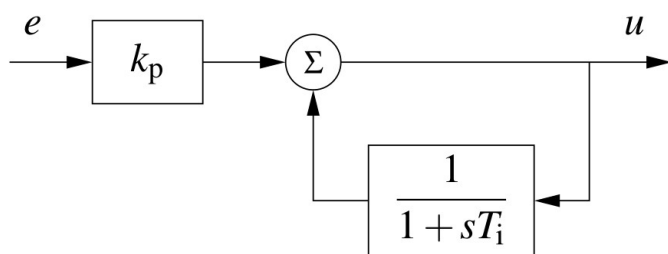
derivative term amplifies noise

\* can become large, saturate actuators, cause instability

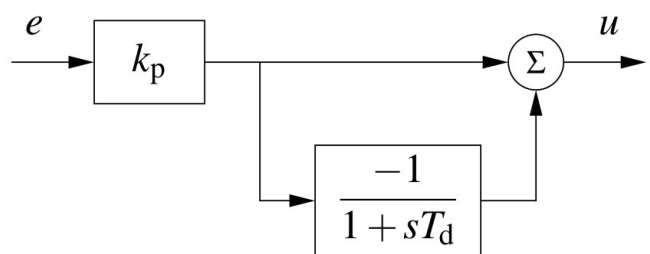
(ii) if we chose PID control to avoid relying on a model, how do we choose gains  $k_p, k_I, k_D$ ?

(i) practically speaking, implementing actual integrator/differentiator is a bad idea

— can instead use low-pass filters in feedback or feedforward



(a) Integral action (automatic reset)



(b) Derivative action

**Figure 11.3:** Implementation of integral and derivative action. The block diagram in (a) shows how integral action is implemented using *positive feedback* with a first-order system, sometimes called automatic reset. The block diagram in (b) shows how derivative action can be implemented by taking differences between a static system and a first-order system.

→ compute transfer function for (a), (b)

$$(a) \text{ Que: } u = k_p e + \frac{1}{1+sT_I} u$$

$$\Leftrightarrow u \left( 1 - \frac{1}{1+sT_I} \right) = k_p e$$

$$\Leftrightarrow u \left( \frac{sT_I}{1+sT_I} \right) = k_p e$$

$$\Leftrightarrow u = \underbrace{k_p \left( 1 + \frac{1}{sT_I} \right)}_{= G_{ue}} e$$

$$= k_p e + \frac{k_I}{s} e, \quad k_I = \frac{k_p}{T_I}$$

$$(b) \text{ Que: } u = k_p e - \frac{k_p}{1+sT_D} e$$

$$= k_p \left( 1 - \frac{1}{1+sT_D} \right) e$$

$$= k_p \left( \frac{sT_D}{1+sT_D} \right) e$$

$$\underbrace{\hspace{10em}}_{= G_{ue}}$$

note: implementation in (a) via low-pass filter feedback yields integral control, but implementation in (b) via low-pass filter feedforward yields filtered derivative control, i.e. derivative control based on filtered error signal

(ii) to tune PID controller gains:

- you can use a model for the system,  
e.g. write down the DE and  
estimate free parameters,  
compute controller gains
- o empirically estimate transfer  
function's Bode plot, apply  
frequency-domain design methods

OR:

- tune the gains until it works!
- o the first (and still most widely-known)  
rules were developed by Ziegler & Nichols  
in the 1940s
- heuristic rules hand-designed to  
work on systems Z & N had access to
- assumes (i.e. is only guaranteed to work)  
for systems with transfer function  
 $\frac{e^{-sT}}{(s+a)}$ , i.e. one pole at  $-a$   
& delay of duration  $T$
- a critical step leverages the Nyquist  
plot & stability criterion:
- 1. set  $K_I, K_D = 0$



2. increase  $k_p$  until system oscillates  
at gain  $k_c$  w/ period  $T_c$
  3. Nyquist implies loop transfer function  
 $L = k_c P(s)$  passes through critical  
point  $-1 \in \mathbb{C}$  at frequency  $\omega_c = \frac{2\pi}{T_c}$
- \* this knowledge of Nyquist plot is  
used to select controller gains:

Type	$k_p$	$T_i$	$T_d$
P	$0.5k_c$		
PI	$0.4k_c$	$0.8T_c$	
PID	$0.6k_c$	$0.5T_c$	$0.125T_c$