1-feedback-principles

[AMV2 Ch 2]

goal: introduce fundamental uses and properties of feedback

topics:

1º. mathematical models of systems

1! differential equations (DE)

12. transfer functions

13. block diagrams

2° effects of feedback

2! disturbance attenuation

2º mmodeled dynamics

23 reference tracking

xread [AMV2 ch 2.2.5] to lean how positive feedback used in digital systems

1º. mathematical models of systems

· we will work with several representations

of linear control systems:

1! differential equations each has

12. transfer functions & provides

13. block diagrams insight

) insight 13. block diagrams * what is a system () ex: consider on RLC circuit 1'. differential equation v(x) = v(x) = v(x) + v(x) + v(x) = v(x) + v(x) + v(x) = v(x) =

* how does input voltage re relate to asput charge y.

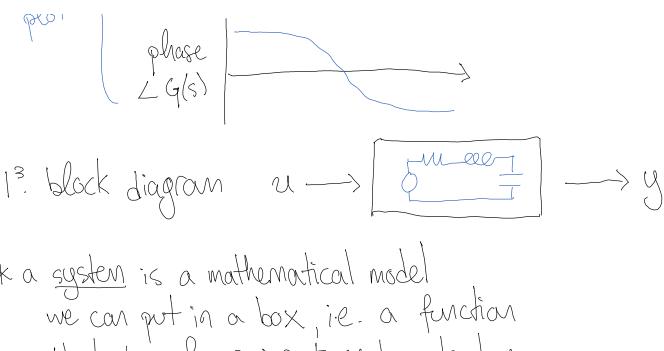
$$(kv) \Rightarrow u = Ri + L \frac{d}{dt}i + \frac{1}{c}y$$

$$= R \frac{d}{dt}y + L \frac{d^2}{dt}y + \frac{1}{c}y$$

12. transfer function

$$u = e^{st} \Rightarrow y = G(s) u = \left(\frac{1}{Ls^2 | Rs + \frac{1}{c}}\right) u$$

$$gain$$



* a system is a mathematical model that transforms input u to output y

1'. (linear) differential equation (DE): [AMV2 Ch 2] $\frac{dt_n}{dt_n} + \alpha_1 \frac{dt_{n-1}}{dt_{n-1}} + \cdots + \alpha_n$ [Nise Ch 3,4] $= p' \frac{q_{r-1}}{q_{r-1}} n + \cdots + p^{n} n$ where u is input y is output t is time and {ap}, {bp} = 1K (i.e. the agis & bg's are real numbers)

- note that (DE) is specified

by two polynamial expressions,

- a(s) = s" + a, s" - 1 + - - + an

b(s) = b, s" - 1 + - - + bn

> called the characteristic polynamial

* these polynamials, and their algebraic

properties, tell us a lot about (DE)

- a "solution" to (DE) is a pair of

signals u: IR -> IR y: IR -> IR

: t >> u(t)

L> ie. smooth functions of time

that satisfy (DE) at all times tell

-> if u, y solve (DE), show that

u': R -> R

: t -> u(t+z)

solve (DE) as well

-> if (u, yi), (u2, y2) solve (DE),

show that (u, + x u2, y, + x y)

solve (DE) as well, where x E R

-> thus (DE) is linear

(DE) is linear & time-invariant (LTI) fact: every solution to (DE) is a linear combination of: the homogeneous solution (i.e. u=0) ¿ a particular solution (u ≠0) fact: when u=0, the solution to $\frac{dt_n}{dt_n} + \alpha_1 \frac{dt_{n-1}}{dt_{n-1}} + \cdots + \alpha_n = 0$ is a linear combination of complex exponentials $y(t) = c_i e^{s_i t} + \cdots + c_n e^{s_n t}$ = E CK e Skt assuming they
They are distinct where $\{s_k\}_{k=1}^n$ are the roots of the characteristic polynomial a(s), 1.e. $Q(S_k) = \tilde{O}$ for $k = 1, \dots, n$ and the coefficients { ck} are determined by the initial condition y(0), y(0), y(0), --, din y(0) $= \left\{ \frac{d^{k}}{d^{k}} y(0) \right\}_{k=1}^{N}$ fact: since ax's are real, the roots are: real -or- complex carjugate pairs - real root Sk ~> real exponential eskt

- real root $S_k \sim real$ exponential $e^{S_k t}$ Ly decays to O(zero) if $S_k < 0$, constant if $S_k = 0$, increases if $S_k > 0$

- complex conjugate pair $s_k = \sigma \pm j \omega \in \mathbb{C}$ \rightarrow est $\cos(\omega t) + j e^{\delta t} \sin(\omega t)$ Ly oscillating within exponential envelope determined by $\sigma \in \mathbb{R}$

12 transfer function:

[AMV2 Ch2]

· from a different perspective, a system [Nise Ch 2] transforms input u to adopt y:

o when $u(t) = e^{st}$, $s \notin \{s_k\}$, and system is linear of time-invariant, guess $y(t) = q(s)e^{st}$

L> some frequency-dependent G: C-> C

The following: $\frac{d}{dt}u = se^{st}$, $\frac{d^2}{dt^2}u = s^2e^{st}$, ..., $\frac{d^n}{dt^n}u = s^ne^{st}$ $\frac{d}{dt}y = sG(s)e^{st}$, ..., $\frac{d^n}{dt^n}y = s^nG(s)e^{st}$

- substituting into (DE) yields $(s^{n} + a, s^{n-1} + \cdots + a_{n}) G(s) e^{st}$ $= (b_1 s^{n-1} + b_2 s^{n-2} + \dots + b_n)$ so $G(s) = b_1 S^{n-1} + b_2 S^{n-2} + \dots + b_n = b(s)$ $s^{n} + a_{1}s^{n-1} + \cdots + a_{N}$ a(s)is the (frequency) transfer function that tells us how (complex) expanential inputs transform into exponential autputs L> recall from signals class that any input can be expressed as a (possibly infinite) sum of complex exponentials via the Fourier transform, * so G(s) contains all the information

summary & synthesis of 1! & 13:

exponential input $u(t) = e^{st}$ to linear system yields exp. at. $y(t) = \sum_{k=1}^{n} c_k e^{skt} + G(s) e^{st}$ Promogreus response to initial state Particular response to initial state

particular response to input signal

· terminology:

- static gain
$$G(0) = \frac{b_n}{a_n} = y$$
 for $u = e^{0 \cdot t} = 1$

- magnitude
$$|z|$$
, organist / phase / ongle $|z|$ of a complex number $|z| = rej\theta$ ore defined by $|z| = r$, $|z| = \theta$

onote: given $u(t) = \sin \omega t = |m| e^{j\omega t}$ $y(t) = |m| (G(j\omega) e^{j\omega t})$

$$= |m(|G(j\omega)|e^{j \angle G(j\omega)}e^{j\omega t})$$

$$= |G(j\omega)| |M e^{j(L G(j\omega) + j\omega k)}$$

-> what happens when $2e(t) = e^{skt}$, $a(s_k) = 0$?

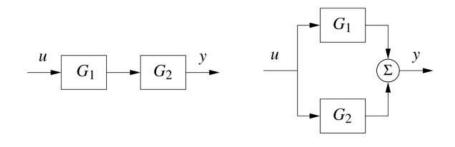
- what does the transfer function tell us?

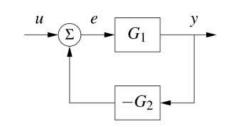
- " (DE) tell us?

_ " (DE) tellus? \rightarrow same Q's for $u(t) = e^{Slt}$, b(sl) = 0> sk termed a pole, se termed a zero ex: consider on RLC circuit (DE) Lig + Rig + C g = V R & Copacitor - con determine transfer function by inspection: $G(s) = \frac{1}{L s^2 + Rs + 1/c}$ \rightarrow show that $G(s) \simeq \left\{ \begin{array}{c} C, s \text{ small} \\ 1/2s^2, s \text{ large} \end{array} \right.$ - what is the interpretation of the circuit's behavior in these two input regimes (small, large)? o given an LTI system u-> [LTI sys] -> y, since $u(t) = e^{st} \rightarrow y(t) = \sum_{k=1}^{n} c_k e^{skt} + q(s) e^{st}$ we want to know whether (hanogenears) encoded

we want to know whether (homogenears) encoded response in CK's is stable, i.e. decays to zero for any initial state -in other words, we want to know if roots of characteristic polynomial $a(s) = s^n + a, s^{n-1} + \cdots + a_n = 0$ have negative real part Lyav know the guadratic formula; analogous formulae exist for cubic & quartic (i.e. 3rd- & 4th-order) polynomials, but generally don't for ligher-order? - Routh (1831-1907) & Hurwitz (1859-1919) found necessary & sufficient criteria for stability using only coefficients {ak} (i.e. not roots {sk}): roots of a(s) have if ord (algebraic condition) negative real part only if (on coeff's {ak}) $Q^{(1)}Q^{(2)} > 0$ $S^2+a_1S+a_2$ $Q_{11}Q_{21}Q_{3}>Q_{1}Q_{1}Q_{2}>Q_{3}$ $5^3 + \alpha_1 S^2 + \alpha_2 S + \alpha_3$ $5^{4} + a_{1}s^{3} + a_{2}s^{2} + a_{3}s + a_{4}$ $\alpha_{11}\alpha_{21}\alpha_{31}\alpha_{4}>0$ $a_1 a_2 > a_3 | a_1 a_2 a_3 > a_1^2 a_4 + a_2^2$ ex: RLC arout L&+R&+t& stable (=) R/1. 1/01 > 0

13. block diagrams pravide a third kind of [AMV2 Ch2] moth model, particularly useful for [Nise Ch 5] specifying system interconnections:





(a)
$$G_{yu}(s) = G_2(s)G_1(s)$$

(b)
$$G_{yu}(s) = G_1(s) + G_2(s)$$

(c)
$$G_{yu}(s) = \frac{G_1(s)}{1 + G_1(s)G_2(s)}$$

- note the convention: Gyu is transfer function from 12 to 19

-> derive formulae for (a), (b), (c) by letting u = est and solving for 4/u

2. effects of feedback

· there are many uses & types of feedback;

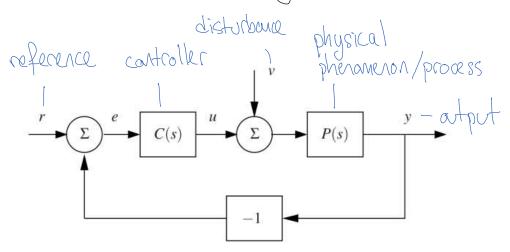
· there are many uses & types of feedback; well focus on these important cases:

- 2! disturbance attenuation
- 2º unmodeled dynamics
- 23 reference tracking

xread [AMV2 ch 2] to lean about other uses & types of feedback

2! disturbance attenuation [AMV2 Ch 2.3]

- consider the block diagram



- for simplicity, consider r = 0(reference tracking studied in later section)

- transfer function from v to y satisfies y = P(v + Ce) e = r - y $= P(v + C(r - y)) \qquad r = 0$

$$= P(N - Cy)$$

$$\Rightarrow (1 + PC)y = PV$$

$$\Rightarrow y = \frac{P}{V}$$

$$1 + PC$$

$$1.e. Gyv(s) = \frac{P(s)}{1 + P(s)C(s)}$$

ex: for simplicity, consider $P(s) = \frac{b}{s+a}$ i.e. y+ay=bu, a,b>0

- velocity of cor;

- angular velocity of flywheel;

- temperature of a mass; - liquid in a reservoir

-> in this case:

r - desired velocity

u - throttle/gas pedal

y - car velocity

v - road slope, Treadwind, ...

a - air resistance, wheel friction ...

b - conversion from force to accel

o try proportional control: u= kpe

o try proportional control:
$$N=kpe$$

i.e. $C(s)=kp$, and hence

 $Gyv=\frac{P}{1+PC}=\frac{b}{1+b}kp$, are negative;

i.e. if all roots of characteristic polynomial

 $G(s)=s+(a+b)kp$, are negative;

i.e. if a+b kp > 0

-in this case, constant disturbance V_s

(e.g. slape of hill) yields

 $Gyv=\frac{P}{1+b}kp$

with time constant $T=\frac{P}{1+b}kp$

with time constant $T=\frac{P}{1+b}kp$

* without feedback $(kp=0)$, $y\rightarrow b$ V_0 /a

of rate $T_0=\frac{P}{1+b}kp$ so feedback boths

i) attenuates disturbance $(a+b)kp$

ii) speeds convergence $(T=\frac{1}{a+b}kp)$ $(T=\frac{1}{a+b}kp)$

otry peoportional—integral control:

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11/1/ 1. pt

 $u(t) = kpe(t) + k_I \int_{-\infty}^{t} e(z) dz$ so transfer function $C(s) = \frac{KPS + K_{\underline{I}}}{}$ = Kp+ KI/S - closing the loop yields $Gyv = \frac{P}{1+PC} = \frac{bs}{s^2 + (a+bkp)s + bk_I}$ -> resily this expression for Gyv -as a (DE) \ddot{y} + $(a+bkp)\ddot{y}$ + $bk_{I}y$ = $b\ddot{v}$ x constant disturbance $v = v_0$ yields zero steady-state error, Gyu(0) = 0? -> what assumption is needed on kp, kI? - to ensure closed-loop system is stable à performance is désirable, we can ture kp, kj: - the characteristic equation of (DE) is Γ $acl(s) = s^2 + (a + bkp) s + b$ d = "besired"L> cl = "closed loop" - if we desire complex-conjugate roots - of ± jw1, - it we desire complex-compare 100005 - od - j wd, $(s + 6d + j \omega_d)(s + 6d - j \omega_d) = s^2 + 2 \delta_d s + \delta_d^2 + \omega_d^2$ yielding solutions est sin(wit), est cos(wit) to the homogeneous equation, i.e. damped oscillations - matching coeff's in char. poly.'s yields $K_p = 26d - a$, $K_T = 6d^2 + \omega d^2$ * let's try a common (and clever) parameter, jation $\omega_{c} = \sqrt{\sigma_{l}^{2} + \omega_{l}^{2}}, \quad \varsigma_{c} = \frac{\sigma_{l}}{\omega_{c}}$ natural frequency damping ratio > this may seem arbitrary - its utility will become more clear as we learn more - | Sc | & | determines response shape, $-\text{Now Gyv(S)} = \frac{P(S)}{1 + P(S)C(S)} = \frac{11 \text{ speed}}{5^2 + 2C_c\omega_c S + \omega_c^2}$ - conclude that disturbances attenuated if the gain | Gyv(jw) | small for all w $|Qyv(jw)| \sim \begin{cases} bw/wc, & w \text{ small} \\ b/w, & w \text{ large} \end{cases}$ note: $\max |Gyv(j\omega)| = |Gyv(j\omega_c)|$

note:
$$\max | (qyv(jw))| = | (qyv(jwc))|$$

$$= \frac{b}{(2 \leqslant cwc)}$$
so w_c large attenuates disturbances

22. unnodeled dynamics

i.e. disturbance [AMV2 Ch 2.4]

- o preceding analysis suggests performance can be arbitrarily good, but reality is not the same as our simple model...
 - Since $k_p = (25c\omega_c a)/h_1 k_I = \omega t/b_1$ large we is high performance, but large kp, KI
 - suppose unnodeled dynamics of sensors, actuators, etc have time constant T>0

$$P(s) = \frac{b}{(s+a)(1+sT)}$$

- now the closed-loop characteristic poly. is $Acl = S(sta)(1+sT) + kps + k_I$

$$= 5^{3}T + 5^{2}(1+aT) + 25cwcs + wc^{2}$$

-> R-H implies closed-loop system stable if wit < 29cwc(1+aT) (=> wit < 29c(1+aT) * conclude that ce is limited by T, i.e.
characteristic time constant of unmodeled dynamics
generally limit achievable performance

I although we rely an simple models for control
design, the resulting controller must always be
sonity-checked / validated on physical system

23. reference tracking

[AMV2 Ch2.5]

- o now consider the case where reference r \$0, e.g. cruise control, satellite tracking
 - suppose plant/process is first-order, $P(s) = \frac{b}{s+a}$ controller is proportional-integral, $C(s) = \frac{b}{s+k_{I}}$
 - for simplicity, neglect disturbance: v=0
 - block diagram algebra yields $Gyr(s) = \frac{P(s)C(s)}{1 + P(s)C(s)} = \frac{bkps + bk_{I}}{s^{2} + (a+bkp)s + bk_{I}}$
 - since Gyr(0)=1, then r constant => y=r (assuming system is stable)
 - to check stability, apply R-H to closed-loop chor. poly. $a_{CR}(s) = s^2 + (a + bkp)s + bk_I$
 - -identifying coefficients with $s^2 + 2s_c \omega_c s + \omega_c^2$ yields $k_p = 2s_c \omega_c - a$, $k_I = \frac{\omega^2}{L}$

yields $k_p = 2c_c \omega_c - a$, $k_T = \frac{\omega^2}{b}$ - with this parameterization, $Gyr(s) = \frac{P(s)C(s)}{1+P(s)C(s)} = \frac{(2c_c \omega_c - a)s + \omega_c^2}{s^2 + 2c_c \omega_c s + \omega_c^2}$ * if $s = j\omega$ and $|\omega| < \omega_c$, then $Gyr(s) \simeq 1$ so ω_c limits bandwidth of references that can be tracked accurately