

Chapter 13

Dynamical Systems and Chaos

13.1 Introduction to Dynamical Systems; Phase Portraits

- 11/27:
- **Dyanmical system:** A system of first-order ODEs.
 - Example: Flows on a line.

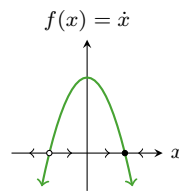


Figure 13.1: Dynamical flows on a line.

- Consider

$$\dot{x} = -x^2 + 4$$

- Graph $f(x) = \dot{x}$, as above.
- When the graph is negative, a particle on the line heads to the left; when it is positive, the particle heads to the right. We indicate this with arrows.
- Then we indicate **fixed points** with circles, **unstable** ones with unfilled circles and **stable** ones with filled circles.
- How do we determine fixed points and stability mathematically?
- Fixed points: Solving $\dot{x} = 0$ yields $x^* = \pm 2$ as fixed points.
- Stability: Consider a point a small distance away from x^* at $x = x^* + \xi$.
 - Approximate \dot{x} near x^* via

$$\dot{x} = f(x) = f(x^* + \xi) \approx f(x^*) + \left. \frac{\partial f}{\partial x} \right|_{x^*} \xi + O(\xi^2)$$

- Then since $f(x^*) = 0$ and we neglect $O(\xi^2)$ for small ξ , we have that

$$\dot{x} = \dot{\xi} = \left. \frac{\partial f}{\partial x} \right|_{x^*} \xi$$

- Looking at Figure 13.1, we can see that the fixed point is stable if $\left. \partial f / \partial x \right|_{x^*} \xi < 0$ and unstable if $\left. \partial f / \partial x \right|_{x^*} \xi > 0$.

- **Fixed point:** A point at which $\dot{x} = 0$.
- **Unstable** (fixed point): A fixed point with the flow heading away from it.
- **Stable** (fixed point): A fixed point with the flow heading toward it.
- Let's promote ourselves up a dimension to the 2D phase plane.
- Example: Pendulum.

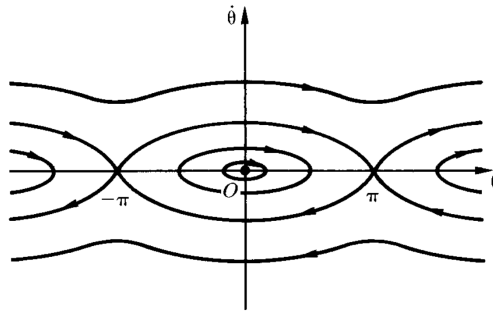


Figure 13.2: Dynamical flows of a pendulum.

- Recall that the Hamiltonian for such a system is

$$H = \frac{p_\theta^2}{2m\ell^2} - mg\ell \cos \theta$$

- Thus, Hamilton's equations are

$$-\dot{p}_\theta = \frac{\partial H}{\partial \theta} = mg\ell \sin \theta \qquad \dot{\theta} = \frac{\partial H}{\partial p_\theta} = \frac{p_\theta}{m\ell^2}$$

- This gives us a system of first-order ODEs.
- Fixed points: $\dot{\theta} = 0$ implies $p_\theta = 0$, implies $\dot{p}_\theta = 0$, implies $\sin \theta = 0$ implies $\theta = 0, \pm\pi, \dots$
- We may now draw a **phase portrait**.
- We get circles corresponding to the switch between momentum and potential energy.
- At the fixed points, we have a special **separatrix**; the particle takes an infinite amount of time to get to the fixed point with unstable equilibrium.
- Then the paths at the top and bottom are other trajectories corresponding to swinging all the way around in one direction or another.
- It is traditional to call these paths *trajectories*, even though they are not physical trajectories $x(t)$.
- **Phase portrait:** A plot that gives the paths of particles at all times.
 - What you gain from a phase portrait is all of the paths, but what you lose is all of the dynamical information (i.e., you have no idea how fast anything is going).
- Linear stability in 2D.
 - In general, we have a system of two first-order ODEs as follows.

$$\begin{cases} \dot{x} = f(x, y) \\ \dot{y} = g(x, y) \end{cases}$$

- Let (x^*, y^*) be a fixed point.
- Then, Taylor expanding, we get

$$\dot{x} = f(x^* + \xi, y^* + \eta) \approx f(x^*, y^*) + \left. \frac{\partial f}{\partial x} \right|_{x^*, y^*} \xi + \left. \frac{\partial f}{\partial y} \right|_{x^*, y^*} \eta + O(\xi^2, \eta^2)$$

$$\dot{y} = g(x^* + \xi, y^* + \eta) \approx g(x^*, y^*) + \left. \frac{\partial g}{\partial x} \right|_{x^*, y^*} \xi + \left. \frac{\partial g}{\partial y} \right|_{x^*, y^*} \eta + O(\xi^2, \eta^2)$$

- From here, we obtain a matrix of coefficients called the **Jacobian matrix**, J , as follows.

$$\begin{pmatrix} \dot{x} \\ \dot{y} \end{pmatrix} = \begin{pmatrix} \dot{\xi} \\ \dot{\eta} \end{pmatrix} = \begin{pmatrix} \frac{\partial f}{\partial x} & \frac{\partial f}{\partial y} \\ \frac{\partial g}{\partial x} & \frac{\partial g}{\partial y} \end{pmatrix} \begin{pmatrix} \xi \\ \eta \end{pmatrix}$$

- The directions of exponential growth and decay occur in the eigendirections of the Jacobian matrix!
- Indeed, in these 2D systems, we can classify the fixed point based on the eigenvalues of J .
- Solve for the eigenvalues using the following formula.

$$\lambda_{1,2} = \frac{1}{2} \left[\text{tr}(J) \pm \sqrt{\text{tr}(J)^2 - 4 \det(J)} \right]$$

- For stability, we need the real parts of both eigenvalues to be less than zero.
- There are three important classifications of such systems.

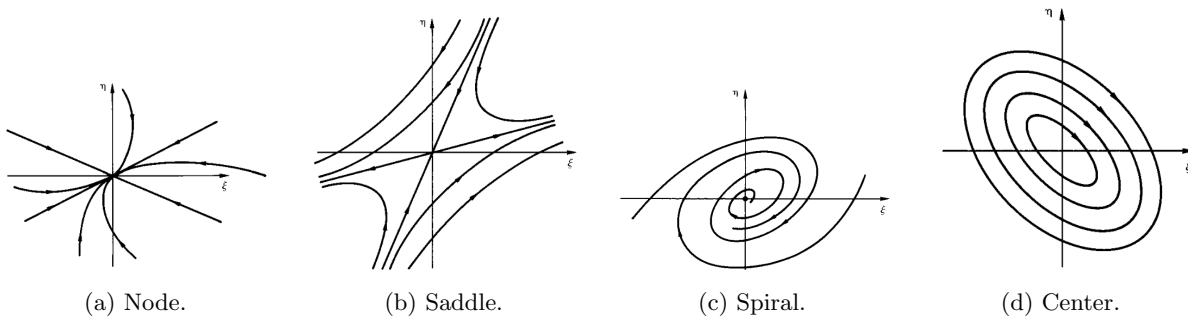


Figure 13.3: Dynamical flows of a 2D system.

1. **Nodes** happen when both λ_1, λ_2 are real and both are positive *or* both are negative.
 - Everything falls into the fixed point in the case $\lambda_1, \lambda_2 < 0$; some things directly (along eigendirections) and other things along curved paths.
 - Alternatively, if $\lambda_1, \lambda_2 > 0$, then everything gets blown away.
 2. If one is greater than zero and one is less than zero, we get a **saddle** point.
 3. If there are some imaginary parts, we get circulation and spiraling. From the eigenvalue formula, we can see that $\lambda_1, \lambda_2 = a \pm bi$ are complex conjugates.
 - If real parts are negative, we spiral inwards; if positive, we spiral outwards.
 - There's also the concept of a **center**; when λ_1, λ_2 are purely imaginary, we get pure circulation where things choose their orbit and stay on it. This is also *stable*, even though things don't fall into the node.
- A handy picture to help us classify any fixed point we want in two dimensions.
 - If we look at systems defined in terms of their trace and determinant, there is a sideways parabola defined by the discriminant of the eigenvalue formula, i.e., via $\text{tr}(J)^2 - 4 \det(J) = 0$.
 - Various paths live in different parts of the map.

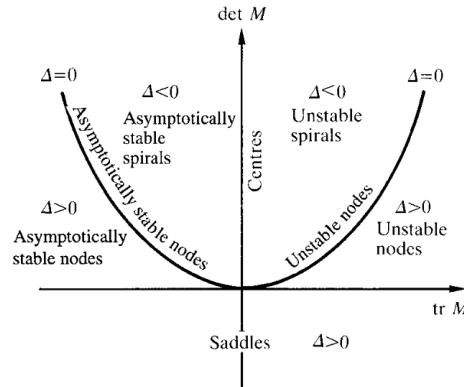


Figure 13.4: Classifying fixed points of a 2D system.

13.2 Bifurcations; Order and Chaos in Hamiltonian Systems

11/29:

- Outline.
 - Bifurcations.
 - Integrability and chaos.
- Today.
 - Ambitious plan:
 - Dynamical systems and phase portraits.
 - What bifurcations are and why they're interesting.
 - When a system is ordered or chaotic.
- Recap.
 - Definition of a **dynamical system**.
 - Recall that Hamilton's equations can help us describe motion in a phase plane through a phase portrait.
 - Essentially, the system of equations defines a vector field (\dot{q}, \dot{p}) at each point (q, p) in the plane. Moreover, trajectories run tangent to vectors.
 - We also saw last time that near a fixed point, for a 2D system $\dot{x} = f(x, y)$ and $\dot{y} = g(x, y)$ such that there exists a point (x^*, y^*) such that $f(x^*, y^*) = g(x^*, y^*) = 0$. Then if we perturb a bit away from this point, our perturbation is given by the Jacobian matrix formula.
 - It follows that we can classify fixed points based on the eigenvalues of the Jacobian, which are given by $\lambda_{1,2} = (\text{tr}(J) \pm \sqrt{\text{tr}(J)^2 - 4 \det(J)})/2$.
 - Recall Figure 13.4.
- Why do the eigenvalues of the Jacobian matrix control the fixed point?
 - Let

$$\vec{v} = \begin{pmatrix} \xi \\ \eta \end{pmatrix}$$

so that

$$\dot{\vec{v}} = J\vec{v}$$

- Diagonalize J to $J = R^{-1}DR$. Then

$$\begin{aligned}\dot{\vec{v}} &= R^{-1}DR\vec{v} \\ R\dot{\vec{v}} &= DR\vec{v} \\ \frac{d}{dt}(\underbrace{R\vec{v}}_{\mu}) &= D(\underbrace{R\vec{v}}_{\mu}) \\ \dot{\mu} &= D\mu\end{aligned}$$

uncouples into

$$\begin{aligned}\dot{\mu}_1 &= \lambda_1\mu_1 & \dot{\mu}_2 &= \lambda_2\mu_2 \\ \mu_1 &= Ae^{\lambda_1 t} & \mu_2 &= Ae^{\lambda_2 t}\end{aligned}$$

- Now let's talk about classifying fixed points in the context of conservative Hamiltonian systems with one degree of freedom.

- In this case, we have

$$p = m\dot{x} \qquad H = \frac{p^2}{2m} + V(x)$$

- Hamilton's equations then give us

$$\dot{x} = \frac{\partial H}{\partial p} = \frac{p}{m} \qquad \dot{p} = -\frac{\partial H}{\partial x} = -\frac{dV}{dx}$$

- Now define

$$f(x, p) = \dot{x} = \frac{p}{m} \qquad g(x, p) = \dot{p} = -\frac{dV}{dx}$$

- Thus, the Jacobian matrix is

$$J = \begin{pmatrix} 0 & \frac{1}{m} \\ -V''(x) & 0 \end{pmatrix}$$

with

$$\text{tr}(J) = 0 \qquad \det(J) = \frac{V''(x)}{m}$$

- Thus, according to Figure 13.4, if $V''(x) > 0$, we get a center, and if $V''(x) < 0$, we get a saddle.
- More specifically, if $V''(x) > 0$, then from the eigenvalues formula,

$$\lambda_{1,2} = i\omega$$

where

$$\omega = \sqrt{\frac{V''(x)}{m}}$$

- Recall the pendulum picture, Figure 13.2.
- In this conservative system, we have a fixed energy $E = p^2/2m + V(x)$. All of the trajectories in Figure 13.2 are level sets of E . So we pick our energy, and the $p(x) = \pm\sqrt{2m(E - V(x))}$, so you can plug in your favorite V , and you will get p .

- Now let's talk about **bifurcations**.
- One of the nice things that this dynamical systems picture gives us is an idea of when a system is going to *really* change.

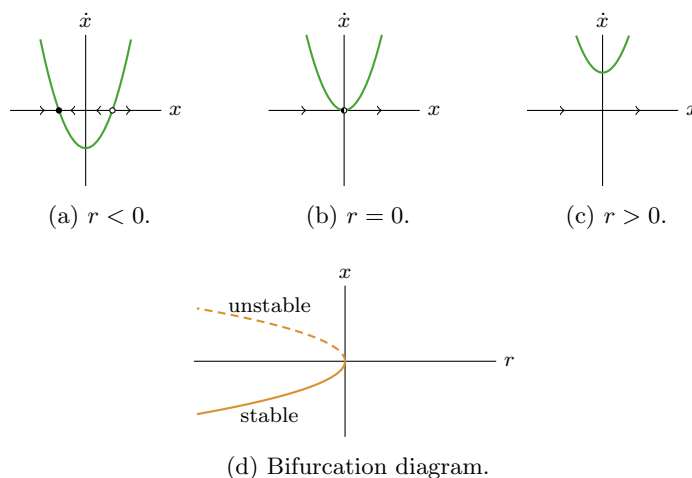


Figure 13.5: Saddle-node bifurcation.

- **Bifurcation:** A number or type of fixed point changes.
- Prototypical type of bifurcation: A **saddle-node** bifurcation.
 - One-dimensional example: Consider $\dot{x} = r + x^2$.
 - As r varies from negative to zero to positive, we get the Figure 13.5a-13.5c. First two fixed points, then one, then none.
 - A **bifurcation diagram** plots r vs. the x -position of the fixed points, stable ones with solid lines and unstable ones with dotted lines. See Figure 13.5d.
- There's a lovely taxonomy of many types of bifurcations. We don't have time to go into them, but here's another one that comes up a lot in physics (we'll talk about it further in discussion section).
- The **pitchfork** bifurcation.

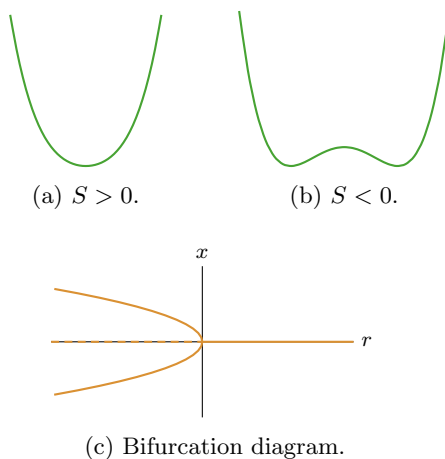


Figure 13.6: Pitchfork bifurcation.

- Consider a potential well of the form

$$V(x) = \frac{S}{2}x^2 + ux^4$$

- There are two important cases here.
 1. If $S > 0$, we get a single well (see Figure 13.6a).
 2. If $S < 0$, then the well divides into two (see Figure ??).
- The particle always wants to slide toward the minimum potential energy, so in the first case, we have one stable branch, and in the second case, we develop three stable branches and one unstable branch. See Figure 13.6c.
- We now return to rigid-body rotation.

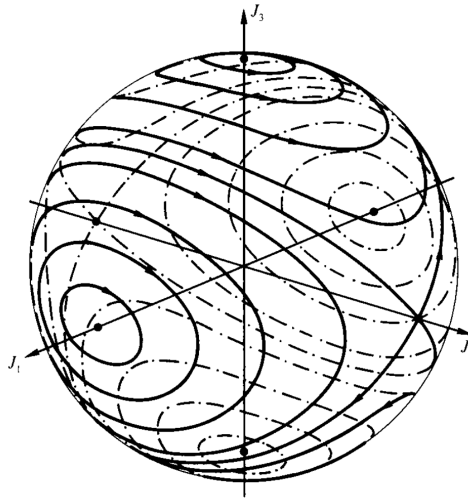


Figure 13.7: Dynamical flows of a rotating rigid body.

- We'll look at this problem more in the last problem of PSet 7.
- Recall the following equation for a rigid body with no external torque.

$$\underbrace{I_1 \dot{\omega}_1}_{\dot{J}_1} + (I_3 - I_2)\omega_2\omega_3 = 0$$

- This equation can be rewritten in the form

$$\begin{aligned} \dot{J}_1 - \frac{I_2 - I_3}{I_2 I_3} J_2 J_3 &= 0 \\ \dot{J}_2 - \frac{I_1 - I_3}{I_1 I_3} J_1 J_3 &= 0 \\ \dot{J}_3 - \frac{I_2 - I_1}{I_1 I_2} J_2 J_1 &= 0 \end{aligned}$$

- We have the conservation laws

$$J_1^2 + J_2^2 + J_3^2 = J^2 \qquad \frac{J_1^2}{I_1} + \frac{J_2^2}{I_2} + \frac{J_3^2}{I_3} = 2T$$

- Thus the fixed points arise as points on a unit sphere where one axis has unit value and the other two have none. These represent rotation about each fixed axis.
- As we'd expect from the tennis racket theorem, the largest ones are stable, and the remaining intermediate axis has saddle points which defines separatrices that interpolate between the other centers.

- Fixed points: $J_3^2 = J^2$, $J_1, J_2 = 0$.
- We now try to understand the possible types of long-term behavior in a system.
- Here are the options.
 1. Flow to a fixed point (this is an equilibrium situation, common especially in system with dissipation).
 2. Systems can oscillate forever (two common cases are centers, which often arise in Hamiltonian systems with energy conservation [planets, pendulums, etc.] and limit cycles of nonlinear systems).
 3. Strange attractor (leads to **chaos**).
- The most canonical set of equations that display chaos (though many systems due this in certain parameter regimes) is the **Lorentz system**, an extremely simplified model of fluid convection between parallel plates at different temperatures.
- **Lorentz system**:

$$\dot{x} = \sigma(y - x) \qquad \dot{y} = \rho x - y - xz \qquad \dot{z} = -\beta z + xy$$

- We watched a [video](#) in class.
- We get attraction to a certain manifold.
- There's a picture in the textbook.
- Characteristics of chaotic systems.
 1. Aperiodic long-term behavior in a deterministic system.
 2. Sensitive dependence on initial conditions.
 - This means that if you have two different trajectories in the phase plane that are separated by distance d_0 at time $t = 0$, then the separation d at time t is exponentially dependent on time via $d = d_0 e^{\lambda t}$ where λ is a **Lyapunov exponent**.
- Hamiltonian systems.
 - n generalized coordinates.
 - Phase space: $2n$.
 - Number of conserved quantities: k .
 - Dimension of the restricted space is $M = 2n - k$.
 - If $k \geq n$, the system is **integrable** and there is no chaos, whereas if $k < n$, then there will be chaos in *some* parameter.
 - The damped, forced pendulum; top with external torques; etc. fall in this regime, so this kind of motion is not hard to find.

13.3 Final Exam Review

12/1:

- Announcements.
 - Thanks for a great quarter!
 - Fill out course evals.
- Our problem.
 - We have N particles with initial positions $\vec{x}_1(0), \dots, \vec{x}_N(0)$ in 3D space and initial momenta $\vec{p}_1(0), \dots, \vec{p}_N(0)$.

- We want to know what they will do, i.e., what is $\vec{x}_i(t)$ for all $i = 1, \dots, N$ and $t > 0$.
- 2-body systems (Chapter 7).
 - The center of mass \vec{R} and relative position \vec{r} are given by

$$\vec{R} = \frac{m_1 \vec{r}_1 + m_2 \vec{r}_2}{m_1 + m_2} \qquad \vec{r} = \vec{r}_1 - \vec{r}_2$$

- We also define the total mass M and reduced mass μ by

$$M = m_1 + m_2 \qquad \mu = \frac{m_1 m_2}{m_1 + m_2}$$

- Under the constant force of gravity...

- The Lagrangian for the system is

$$L = \frac{1}{2} M \dot{\vec{R}}^2 + M \vec{g} \cdot \vec{R} + \frac{1}{2} \mu \dot{\vec{r}}^2 - V_{\text{int}}(\vec{r})$$

- The EOMs separate into

$$\begin{aligned} M \ddot{\vec{R}} &= M \vec{g} \\ \mu \ddot{\vec{r}} &= \vec{F}_{12} \end{aligned}$$

where the second equation above is that of a one-body problem.

- We then solve for \vec{r}, \vec{R} and use these to find \vec{r}_1, \vec{r}_2 via

$$\begin{aligned} \vec{r}_1 &= \vec{R} + \frac{m_2}{M} \vec{r} \\ \vec{r}_2 &= \vec{R} - \frac{m_1}{M} \vec{r} \end{aligned}$$

- It is often useful to do this in the center of mass frame (*).

- We convert into this frame with

$$\begin{aligned} \vec{r}_1^* &= +\frac{m_2}{M} \vec{r} \\ \vec{r}_2^* &= -\frac{m_1}{M} \vec{r} \end{aligned}$$

- In the CM frame, we have the following laws.

$$\vec{P}^* = 0 \qquad \vec{J}^* = \mu \vec{r} \times \dot{\vec{r}} \qquad T^* = \frac{1}{2} \mu \dot{\vec{r}}^2$$

- In the lab frame, we have the following laws.

$$\vec{P} = M \dot{\vec{R}} \qquad \vec{J} = M \vec{R} \times \dot{\vec{R}} + \vec{J}^* \qquad T = \frac{1}{2} M \dot{\vec{R}}^2 + T^*$$

- Moving onto many-body systems (Chapter 8).

- We generalize the center of mass to

$$\vec{R} = \frac{1}{M} \sum_{\alpha} m_{\alpha} \vec{r}_{\alpha}$$

where $M = \sum_{\alpha} m_{\alpha}$.

- Within the CM frame (*),

$$\vec{r}_\alpha = \vec{R} + \vec{r}_\alpha^* \quad \vec{J}^* = \sum_\alpha m_\alpha \vec{r}_\alpha^* \times \dot{\vec{r}}_\alpha^* \quad T^* = \sum_\alpha \frac{1}{2} m_\alpha \dot{\vec{r}}_\alpha^{*2}$$

- Within the lab frame,

$$\vec{P} = M\dot{\vec{R}} \quad \vec{J} = M\vec{R} \times \dot{\vec{R}} + \vec{J}^* \quad T = \frac{1}{2} M \dot{\vec{R}}^2 + T^*$$

- For no force or constant force (like gravity), the Hamiltonian and Lagrangian separate into CM and rest of system.
- Such problems are hard to solve in general, so we quickly specified to rigid bodies.
- Rigid bodies (Chapter 9 + a bit of 10).
 - No changes in internal potential energy.
 - We define the angular momentum of the whole system as

$$\vec{J} = \overleftrightarrow{I} \vec{\omega}$$

where \overleftrightarrow{I} is the moment of inertia tensor and $\vec{\omega}$ is the instantaneous angular velocity.

- There is a special set of coordinates here called the principal axes. In the basis of the principal axes $\hat{e}_1, \hat{e}_2, \hat{e}_3$, the moment of inertia tensor is diagonal, i.e.,

$$\overleftrightarrow{I} = \begin{bmatrix} I_1 & 0 & 0 \\ 0 & I_2 & 0 \\ 0 & 0 & I_3 \end{bmatrix}$$

where

$$I_1 = \int dV \rho_m (y^2 + z^2)$$

for ρ_m the density, and similarly for I_2, I_3 .

- The products of inertia vanish for these axes; we can use this as a check.
- How do we find a moment of inertia?
 - Use the parallel axis theorem, which tells us that

$$I_1 = M(X^2 + Y^2) + I_1^*$$

where I_1^* is I_1 with the origin at the center of mass and X, Y are center of mass coordinates.

- Have Routh's Rules in our mind or in our note's sheet! These are the factors for cylindrical, ellipsoidal, parallelepiped, etc. symmetry. Very important!
- In the principal axis basis,

$$\vec{J} = I_1 \omega_1 \hat{e}_1 + I_2 \omega_2 \hat{e}_2 + I_3 \omega_3 \hat{e}_3$$
- The equations of motion for a rigid body:

$$\dot{\vec{J}} = \sum \vec{r} \times \vec{F} \quad \dot{\vec{P}} = M\ddot{\vec{R}} = \sum \vec{F}$$

- These equations of motion can help us answer questions such as, “why does a spinning top not fall over?”
 - Here, the EOMs are $\dot{\vec{J}} = I_3 \omega \hat{e}_3$ and $\vec{F} = -Mg\hat{k}$.
 - Essentially, we have a torque, which is a change in \vec{J} in the $-\hat{e}_3 \times \hat{k}$ direction.
 - This causes $\vec{\omega}$ to rotate, leading to the phenomenon of precession.

- What balances gravity is $M\ddot{\vec{R}} = \sum \vec{F} = \vec{F}_{\text{pivot}} + \vec{F}_{\text{gravity}}$.
- This is used to solve for a force on the pivot via $m\ddot{\vec{R}} = \dot{\omega} \times \vec{R} + \vec{\omega} \times (\vec{\omega} \times \vec{R})$.
- In contrast, if $\omega = 0$, then $\dot{\vec{J}}$ is in the $-\hat{e}_3 \times \hat{k}$ direction, so the \vec{J} component is into the board, i.e., along \hat{e}_1 .
- Stability: $I_1 < I_2 < I_3$ for a freely rotating body.
 - Rotation about \hat{e}_2 is unstable.
 - Rotation about \hat{e}_1, \hat{e}_3 is stable.
 - Energy in this paradigm.

$$T = \frac{1}{2}I_1\omega_1^2 + \frac{1}{2}I_2\omega_2^2 + \frac{1}{2}I_3\omega_3^2 \qquad T^* = \frac{1}{2}I_1^*\omega_1^2 + \frac{1}{2}I_2^*\omega_2^2 + \frac{1}{2}I_3^*\omega_3^2$$

- In terms of Euler angles, i.e., by writing $\vec{\omega} = \dot{\phi}\hat{k} + \dot{\theta}\hat{e}'_2 + \dot{\psi}\hat{e}_3$, then for a symmetric body,

$$T = \frac{1}{2}I_1\dot{\phi}^2 \sin^2 \theta + \frac{1}{2}I_1\dot{\theta}^2 + \frac{1}{2}I_3(\dot{\psi} + \dot{\phi} \cos \theta)^2$$

implies fixed point.

- With translations, it is usually convenient to separate out this translation from the center of mass, yielding

$$T = \frac{1}{2}M(\dot{X}^2 + \dot{Y}^2 + \dot{Z}^2) + \frac{1}{2}I_1^*(\dot{\phi}^2 \sin^2 \theta + \dot{\theta}^2) + \frac{1}{2}I_3^*(\dot{\psi} + \dot{\phi} \cos \theta)^2$$

- We are not expected to memorize formulas this complicated; if anything this complicated were to arise, she would simply put it on the exam.
- After this, we moved onto...
- Hamiltonian mechanics (Chapter 12).

- We have our friend the Lagrangian $L = T - V$, which gives us generalized momenta

$$p_\alpha = \frac{\partial L}{\partial \dot{q}_\alpha}$$

- From here, we can construct the Hamiltonian

$$H = \sum_\alpha p_\alpha \dot{q}_\alpha - L$$

- For a natural, conservative system: $H = T + V = E$.
- We then get $2n$ first-order ODEs called Hamilton's equations, which are given by

$$-\dot{p}_\alpha = \frac{\partial H}{\partial q_\alpha} \qquad \dot{q}_\alpha = \frac{\partial H}{\partial p_\alpha}$$

- Cool things about the Hamiltonian.
 1. If q_α is not in H , then q_α is ignorable, and p_α is conserved.
 2. Use this to construct effective potential functions via

$$H = \frac{p_\alpha^2}{2m} + U(x, \text{constants})$$

- If we have a function $G(q, p)$, then because H is the time translation operator, we have that

$$\frac{dG}{dt} = \frac{\partial G}{\partial t} + [G, H]$$

- Thus, if $\partial G/\partial t = 0$ and $[G, H] = -[H, G] = 0$, then G is a conserved quantity.
- It follows that the Galilean relativity principles, which require certain symmetries of H , are conserved quantities for an isolated system.
- The quantities conserved are energy, linear momentum, angular momentum.
- Email her if we have questions about dynamical systems.

Final Equations Sheet

Jerison

2 ~~Sanity~~ check Units!

$$p = \dot{r} = Fx$$

3 Conservative force condition: $\vec{F} = -\vec{\nabla} V$, $\vec{\nabla} \times \vec{F} = 0$ Cylindrical coords: $x = r \cos \theta$, $y = r \sin \theta$, $z = z$

$$\text{Torque: } \vec{G} = \vec{r} \times \vec{F} = \dot{\vec{J}}, \quad \vec{J} = m \vec{r} \times \dot{\vec{r}}$$

$$\text{Kepler's 2nd law: } \frac{dA}{dt} = \frac{1}{2} r^2 \dot{\theta} = \frac{J}{2m}; \quad \text{Kepler's 3rd law: } \left(\frac{a}{a_0}\right)^3 = \frac{m}{M} \Rightarrow a^3 = \frac{GM}{m}$$

$$L = T - V, \quad \frac{d}{dt} \left(\frac{\partial L}{\partial \dot{q}_i} \right) = \frac{\partial L}{\partial q_i}, \quad p_i = \frac{\partial L}{\partial \dot{q}_i}, \quad F_i = \frac{\partial L}{\partial q_i}$$

$$5 \quad \frac{d\vec{r}}{dt} = \dot{\vec{r}} + \vec{\omega} \times \vec{r}$$

$$\text{Magnetism / Larmor: } \vec{F} = q \vec{v} \times \vec{B}, \quad \omega_L = \frac{qB}{2m}$$

$$7 \quad \vec{R}, \vec{r}, M, m \propto \frac{m_1 m_2}{m_1 + m_2}, \quad \text{Gravity: } L = \frac{1}{2} M \dot{\vec{R}}^2 + M \vec{g} \cdot \vec{R} + \frac{1}{2} m \dot{\vec{r}}^2 - V_{int}(\vec{r}) \Rightarrow M \ddot{\vec{R}} = M \vec{g}, \quad m \ddot{\vec{r}} = \vec{F}_{12}$$

$$\vec{r}_1 = \vec{R} + \frac{m_2}{M} \vec{r}, \quad \vec{r}_2 = \vec{R} - \frac{m_1}{M} \vec{r}, \quad \vec{r}_1^* = \frac{m_2}{M} \vec{r}, \quad \vec{r}_2^* = -\frac{m_1}{M} \vec{r}$$

$$p^* = 0, \quad J^* = m \dot{\vec{r}} \times \vec{r}, \quad T^* = \frac{1}{2} m \dot{\vec{r}}^2, \quad \vec{p} = M \dot{\vec{R}}, \quad \vec{J} = M \vec{R} \times \dot{\vec{R}} + J^*, \quad T = \frac{1}{2} M \dot{\vec{R}}^2 + T^*$$

$$8 \quad \text{Index by } a: \quad \vec{r}_a = \vec{R} + \vec{r}_a^*, \quad \vec{J}^* = \sum_a m_a \vec{r}_a^* \times \dot{\vec{r}}_a^*, \quad T^* = \sum_a \frac{1}{2} m_a \dot{\vec{r}}_a^{*2}$$

9

$$\begin{bmatrix} J_x \\ J_y \\ J_z \end{bmatrix} = \begin{bmatrix} I_{xx} & I_{xy} & I_{xz} \\ I_{yx} & I_{yy} & I_{yz} \\ I_{zx} & I_{zy} & I_{zz} \end{bmatrix} \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}, \quad \text{In principal axis basis } \{\hat{e}_1, \hat{e}_2, \hat{e}_3\}, \quad \hat{I} = \text{diag}(I_1, I_2, I_3), \quad \vec{J} = \sum_{i=1}^3 I_i \omega_i \hat{e}_i$$

$$I_{zz} = \sum_a m_a \rho_a^2 = \sum_a m_a (x_a^2 + y_a^2) \approx \iiint \rho_m(\vec{r}) (x^2 + y^2) \approx M(x^2 + y^2) + I_{zz}^*$$

$$I_{xz} = -\sum_a m_a x_a z_a \approx -M x z + I_{xz}^*$$

a > b > c

$$\text{Routh's rules: } I_1^* = M(\lambda_y b^2 + \lambda_z c^2), \quad I_2^*(a, b), \quad I_3^*(a, b)$$

$$\cdot \text{Ellipsoid: } 1/6, \quad \text{Parallelepiped: } 1/3, \quad (\text{Cylinder}) \quad x=y=1/4, \quad z=1/2$$

$$\text{EOMs: } \vec{J} = \sum \vec{r} \times \vec{F}, \quad \vec{p} = M \dot{\vec{R}} = \sum \vec{F}$$

Spinning top:

$$I_1 \dot{\omega}_1 + (I_3 - I_1) \omega_1 \omega_3 = G_1$$

Stability: $I_1 < I_2 < I_3$, \hat{e}_2 unstable, \hat{e}_1, \hat{e}_3 stable

$$I_1 \dot{\omega}_2 + (I_1 - I_3) \omega_3 \omega_1 = G_2$$

$$I_1 \dot{\omega}_3 + (I_2 - I_1) \omega_1 \omega_2 = G_3$$

$$\text{Euler angles: } \vec{\omega} = \dot{\theta} \hat{k} + \dot{\phi} \hat{e}_1' + \dot{\psi} \hat{e}_3$$

$$\cdot \text{Symmetric body } (I_1 = I_2): \vec{\omega} = -\dot{\theta} \sin \theta \hat{e}_1' + \dot{\theta} \hat{e}_2' + (\dot{\psi} + \dot{\theta} \cos \theta) \hat{e}_3$$

12 $H = \sum_{\beta=1}^n p_{\beta} \dot{q}_{\beta} - L$, β indexes generalized coordinates, natural conservative system $\Rightarrow H = T + V = E$

Natural (system): L is not a function of t .

Hamilton's eqns: $-p_{\alpha} = \frac{\partial H}{\partial \dot{q}_{\alpha}}$, $\dot{q}_{\alpha} = \frac{\partial H}{\partial p_{\alpha}}$

Ignorable: $q_{\alpha} \notin H \Rightarrow p_{\alpha}$ is conserved $\Rightarrow V_{\text{eff}}$

$G(q, p) \Rightarrow \frac{dG}{dt} = \frac{\partial G}{\partial t} + [G, H]$; can tell you if G is conserved.

$$[F, G] = \sum_{\alpha, \beta} \left(\frac{\partial F}{\partial q_{\alpha}} \frac{\partial G}{\partial p_{\alpha}} - \frac{\partial F}{\partial p_{\alpha}} \frac{\partial G}{\partial q_{\alpha}} \right)$$

13 \circ = unstable, \bullet = stable, $\dot{x} = 0 \Rightarrow$ fixed

$$\begin{pmatrix} \dot{x} \\ \dot{y} \end{pmatrix} = \begin{pmatrix} \frac{\partial f}{\partial x} & \frac{\partial f}{\partial y} \\ \frac{\partial g}{\partial x} & \frac{\partial g}{\partial y} \end{pmatrix} \begin{pmatrix} \xi \\ \eta \end{pmatrix}, \quad \lambda_{1,2} = \frac{1}{2} \left(\text{tr}(J) \pm \sqrt{(\text{tr}(J))^2 - 4 \det(J)} \right)$$

