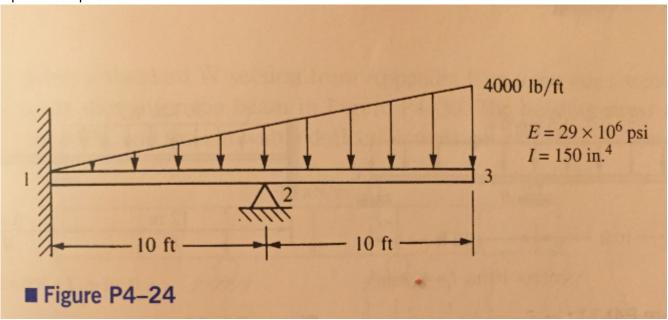
Problem 4.24

Use Mathcad, Matlab, ect to solve Logan Problems. Find the rections, nodal displacement and plot the interpolated displacement for the beams



```
In [1]: # Import necessary packages

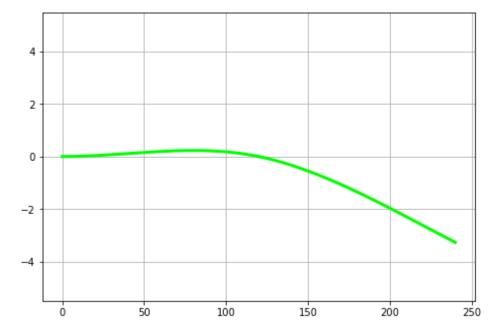
from matplotlib import pyplot as plt
import numpy as np
import sympy as sp
sp.init_printing()
%matplotlib inline
```

```
In [2]: # Initialize problem parameters
         E = 29e6
         I = 150
         L = 10*12
         W = 4000/12
         global stiffness matrix = E*I/sp.Pow(L, 3)*sp.Matrix([[12, 6*L, -12, 6*L, 0, 0])
         ],\
                                               [6*L, 4*sp.Pow(L, 2), -6*L, 2*sp.Pow(L, 2)
         ), 0,0],
                                               [-12, -6*L, 24, 0, -12, 6*L],
                                               [6*L, 2*sp.Pow(L, 2), 0, 8*sp.Pow(L, 2),
         -6*L, 2*sp.Pow(L, 2)],
                                               [0, 0, -12, -6*L, 12, -6*L],
                                               [0, 0, 6*L, 2*sp.Pow(L, 2), -6*L, 4*sp.Po
         w(L, 2)]])
         global_stiffness_matrix
                                              -30208.33333333333
Out[2]:
          30208.33333333333
                                 1812500.0
                                                                    1812500.0
               1812500.0
                                145000000.0
                                                  -1812500.0
                                                                    72500000.0
          -30208.33333333333
                                -1812500.0
                                               60416.6666666667
                                                                        0
                                                                                 -3020
               1812500.0
                                72500000.0
                                                       0
                                                                   290000000.0
                                     0
                   0
                                              -30208.33333333333
                                                                    -1812500.0
                                                                                  3020
                   0
                                     0
                                                   1812500.0
                                                                    72500000.0
In [3]: # Reduce stiffness matrix according to what is constrained
         reduced stiffness matrix = global stiffness matrix[3:, 3:]
         reduced stiffness matrix
Out[3]:
          290000000.0
                           -1812500.0
                                             72500000.0
                                            -1812500.0
           -1812500.0
                        30208.3333333333
           72500000.0
                           -1812500.0
                                            145000000.0
In [4]: # Generate equivalent node forces for distributed loads
         F reduced equivalent nodal forces = sp.Matrix([[-w*sp.Pow(L, 2)/30], \
                                                        [-17*w*L/40],
                                                        [w*sp.Pow(L,2)/15]])
         F reduced equivalent nodal forces
         \lceil -160000.0 \rceil
Out[4]:
           -17000.0
           320000.0
```

```
In [5]: # Calculate displacements that are not constrained
         non_constrained_displacements = reduced_stiffness_matrix.LUsolve(F_reduced_equ
         ivalent nodal forces)
         non constrained displacements
         \lceil -0.0129655172413793 \rceil
Out[5]:
            -3.27724137931034
           -0.0322758620689655
In [6]: # Calculate actual reaction forces
         global_displacements = sp.Matrix([0, 0, 0, non_constrained_displacements])
         effective forces = global stiffness matrix*global displacements
         equivalent_nodal_forces = sp.Matrix([-3*w*L/40,\
                                                -w*sp.Pow(L, 2)/60,
                                                -w*L/2,
                                                F_reduced_equivalent_nodal_forces])
         F = effective forces - equivalent nodal forces
Out[6]:
                    -20500.0
               -859999.999999999
                     60500.0
                        0
            -3.63797880709171 \cdot 10^{-12}
In [7]: # Interpolate from nodal displacements according to shape functions
         def calculate interpolated displacement(x data, length, v 1, phi 1, v 2, phi 2
         ):
             N1 = (1/(L^{**3}))^*(2^*x_data^{**3} - 3^*x data^{**2}L + L^{**3})
             N2 = (1/(L^{**3}))^*(L^*x_data^{**3} - 2^*L^{**2}x_data^{**2} + x_data^*L^{**3})
             N3 = (1/(L^{**3}))^*(-2^*x data^{**3} + 3^*L^*x data^{**2})
             N4 = (1/(L^{**3}))*(L^{*}x data^{**3} - x data^{**2}L^{**2})
             return N1*v_1 + N2*phi_1 + N3*v_2 + N4*phi_2
```

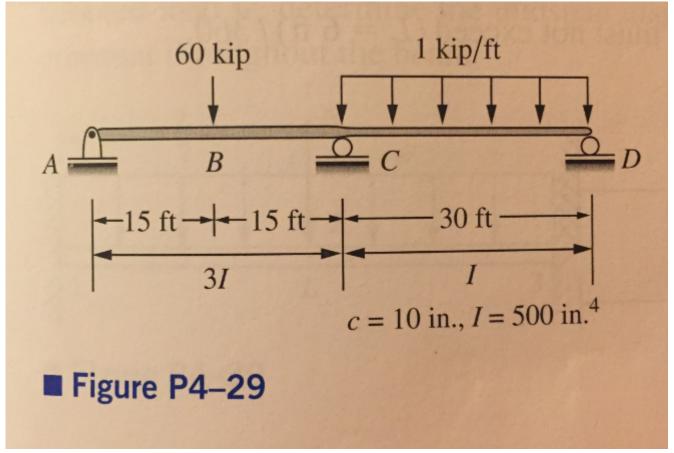
In [8]: # Plot interpolated displacements

```
fig = plt.figure()
ax = fig.add_axes([0, 0, 1, 1])
x = np.arange(0, L, 0.1)
beam_element_1_displacement = calculate_interpolated_displacement(x, L, 0, 0,
0, non_constrained_displacements[0])
beam_element_2_displacement = calculate_interpolated_displacement(x, L, 0, non
_constrained_displacements[0], non_constrained_displacements[1], non_constrain
ed_displacements[2])
linewidth = 3
ax.plot(x, beam_element_1_displacement, color='lime', linewidth=linewidth)
ax.plot(x+L, beam_element_2_displacement, color='lime', linewidth=linewidth)
plt.ylim(-5.5, 5.5)
ax.grid(True)
```



Problem 4.29

Use Mathcad, Matlab, ect to solve Logan Problem 4.29. Find the rections, nodal displacement and plot the interpolated displacement for the beam. Also plot the stress along the beam and confirm with CREO



```
In [9]: # Initialize problem parameters
         E, I, L, P, w = sp.symbols('E, I, L, P, w')
         E = 30e6
         I = 500
         L = 30*12
         P = 60e3
         W = 1000/12
        k_1 = ((E*3*I)/sp.Pow(L, 3))*sp.Matrix([[12, 6*L, -12, 6*L], \])
                                                  [6*L, 4*sp.Pow(L, 2), -6*L, 2*sp.Pow(L
         , 2)],
                                                  [-12, -6*L, 12, -6*L],
                                                  [6*L, 2*sp.Pow(L, 2), -6*L, 4*sp.Pow(L
         , 2)]])
         k_2 = ((E*I)/sp.Pow(L, 3))*sp.Matrix([[12, 6*L, -12, 6*L], 
                                                [6*L, 4*sp.Pow(L, 2), -6*L, 2*sp.Pow(L,
         2)],
                                                [-12, -6*L, 12, -6*L],
                                                [6*L, 2*sp.Pow(L, 2), -6*L, 4*sp.Pow(L,
         2)]])
```

```
In [10]: # Generate global stiffness matrix for this problem
```

```
global_stiffness_matrix = sp.zeros(6, 6)
global_stiffness_matrix[0:4, 0:4] = global_stiffness_matrix[0:4, 0:4] + k_1
global_stiffness_matrix[2:6, 2:6] = global_stiffness_matrix[2:6, 2:6] + k_2
global_stiffness_matrix
```

Out[10]:

```
11574.0740740741
                   2083333.33333333
                                      -11574.0740740741
                                                          2083333.33
2083333.33333333
                      500000000.0
                                      -2083333.333333333
                                                             25000000
-11574.0740740741
                   -2083333.333333333
                                       15432.0987654321
                                                          -1388888.88
2083333.33333333
                      250000000.0
                                      -1388888.88888889
                                                          0
                           0
                                      -3858.02469135802
                                                          -694444.444
       0
                           0
                                       694444.44444444
                                                          83333333.3
```

```
In [11]: # construct reduced stiffness matrix (manually cause I'm running out of time)
         k_reduced = sp.zeros(3, 3)
         k reduced[0, 0] = global stiffness matrix[1, 1]
         k_reduced[0, 1] = global_stiffness_matrix[1, 3]
         k_reduced[0, 2] = global_stiffness_matrix[1, 5]
         k reduced[1, 0] = global stiffness matrix[3, 1]
         k reduced[1, 1] = global stiffness matrix[3, 3]
         k reduced[1, 2] = global stiffness matrix[3, 5]
         k_reduced[2, 0] = global_stiffness_matrix[5, 1]
         k reduced[2, 1] = global stiffness matrix[5, 3]
         k reduced[2, 2] = global stiffness matrix[5, 5]
         k reduced
Out[11]: \[ \int 500000000.0 \]
                            250000000.0
                                                     0
                                            83333333.3333333
           250000000.0 666666666666667
                                            166666666666667
                0
                         83333333.33333333
In [12]: # Generate equivalent node forces for distributed loads
         F_reduced_equivalent_nodal_forces = sp.Matrix([-P*L/8, -w*sp.Pow(L, 2)/12 + P*
         L/8, w*sp.Pow(L, 2)/12])
         F reduced equivalent nodal forces
Out[12]:
          1-2700000.0
            1800000.0
            900000.0
In [13]: # Calculate displacements that are not constrained
         non constrained displacements = k reduced.LUsolve(F reduced equivalent nodal f
         orces)
         non_constrained_displacements
Out[13]:
          \lceil -0.0081 \rceil
            0.0054
            0.0027
```

```
In [14]: # Calculate all displacements
             global_displacements = sp.Matrix([0,\
                                                non_constrained_displacements[0],
                                                non_constrained_displacements[1],
                                                non constrained displacements[2]])
             effective_forces = global_stiffness_matrix*global_displacements
             equivalent_nodal_forces = sp.Matrix([-P/2,
                                                  -P*L/8,
                                                  -P/2 - w*L/2,
                                                  -w*sp.Pow(L, 2)/12 + P*L/8,
                                                  -w*L/2,
                                                  w*sp.Pow(L, 2)/12])
             global_displacements
   Out[14]:
               -0.0081
               0.0027
   In [15]: # Calculate actual reaction forces
             F = effective_forces - equivalent_nodal_forces
   Out[15]:
                        24375.0
               -4.65661287307739\cdot 10^{-10}
                        56250.0
               -2.3283064365387 \cdot 10^{-10}
                         9375.0
R_a = 24.4\, kip R_b = 56.3\, kip R_c = 9.38\, kip
```

In [19]: # Plot interpolated deflections fig = plt.figure() ax = fig.add axes([0, 0, 1, 1])x = np.arange(0, L, 0.1)beam_element_1_displacement = calculate_interpolated_displacement(x, L, 0, non constrained displacements[0], 0, non constrained displacements[1]) beam element 2 displacement = calculate interpolated displacement(x, L, 0, non constrained displacements[1], 0, non constrained displacements[2]) max_displacement = np.amax(np.abs(beam_element_1_displacement)) max test = np.amax(np.abs(beam element 1 displacement)) if max_test > max_displacement: max_displacement = max_test print("max displacement is {} in.".format(max_test)) linewidth = 3ax.plot(x, beam_element_1_displacement, color='lime', linewidth=linewidth) ax.plot(x+L, beam element 2 displacement, color='lime', linewidth=linewidth) ax.set title('Deflection Curve') plt.ylim(-3, 3) ax.grid(True)

max displacement is 0.613458658854167 in.

