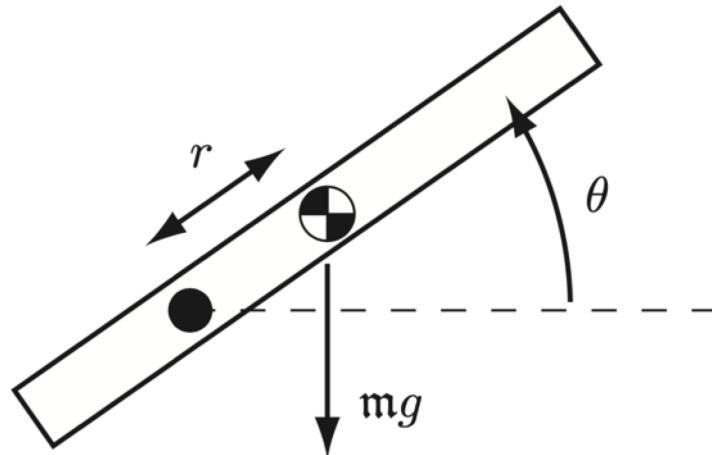


Chapter 2	Configuration Space
Chapter 3	Rigid-Body Motions
Chapter 4	Forward Kinematics
Chapter 5	Velocity Kinematics and Statics
Chapter 6	Inverse Kinematics
Chapter 7	Kinematics of Closed Chains
Chapter 8	Dynamics of Open Chains
Chapter 9	Trajectory Generation
Chapter 10	Motion Planning
Chapter 11	Robot Control
	11.1 Control System Overview
	11.2 Error Dynamics
	11.3 Motion Control with Velocity Inputs
	11.4 Motion Control with Torque or Force Inputs
Chapter 12	Grasping and Manipulation
Chapter 13	Wheeled Mobile Robots

## Important concepts, symbols, and equations

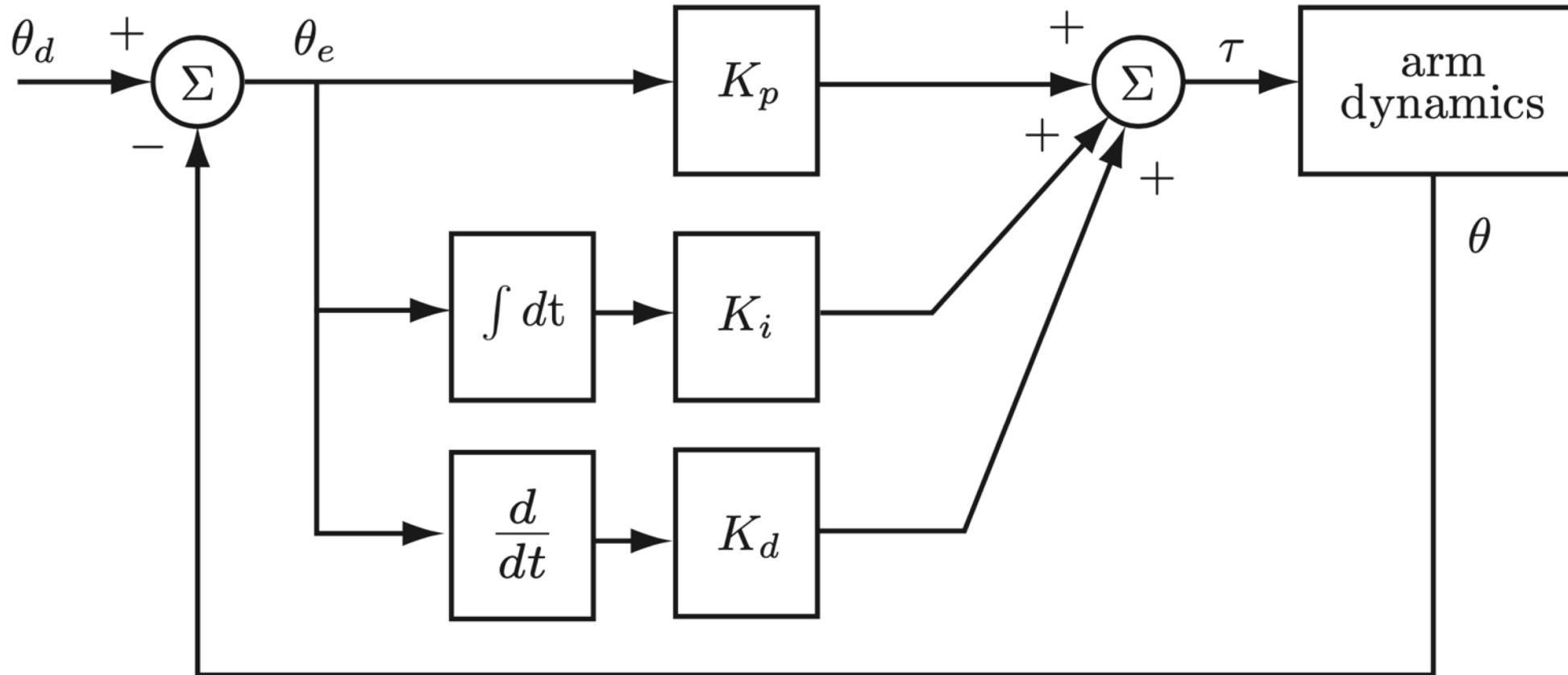


$$\tau = M\ddot{\theta} + \underbrace{m\bar{g}r \cos \theta}_{h(\theta, \dot{\theta})} + b\dot{\theta}$$

## Proportional-Integral-Derivative (PID) control

$$\tau = K_p \theta_e + K_i \int \theta_e(t) dt + K_d \dot{\theta}_e$$

## Important concepts, symbols, and equations (cont.)



## Important concepts, symbols, and equations (cont.)

Setpoint PD control,  $g = 0$

$$\tau = K_p \theta_e + \cancel{K_i} \int \theta_e(t) dt + K_d \dot{\theta}_e$$

$$\tau = M \ddot{\theta} + \cancel{m g r} \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

$$M \ddot{\theta} + b \dot{\theta} = K_p \theta_e + K_d \dot{\theta}_e$$

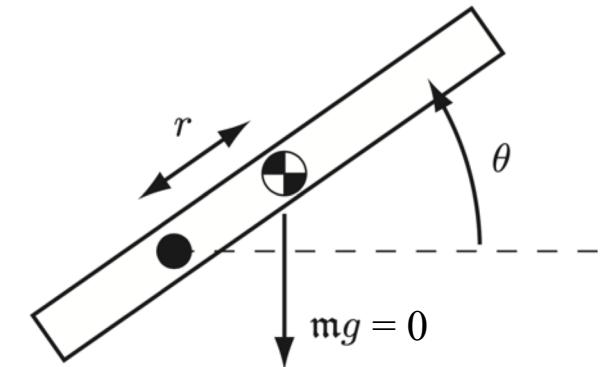
$$\ddot{\theta}_e + \frac{b + K_d}{M} \dot{\theta}_e + \frac{K_p}{M} \theta_e = 0$$

$$\zeta = \frac{b + K_d}{2\sqrt{K_p M}} \quad \omega_n = \sqrt{\frac{K_p}{M}}$$

For what gains  
are the error  
dynamics stable?

$K_d > -b$

$K_p > 0$



## Important concepts, symbols, and equations (cont.)

Setpoint PD control,  $g \neq 0$

$$\tau = K_p \theta_e + \cancel{K_i}^0 \int \theta_e(t) dt + K_d \dot{\theta}_e$$

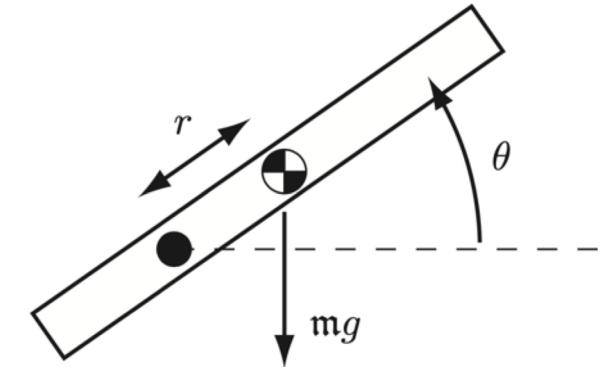
$$\tau = M \ddot{\theta} + m g r \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

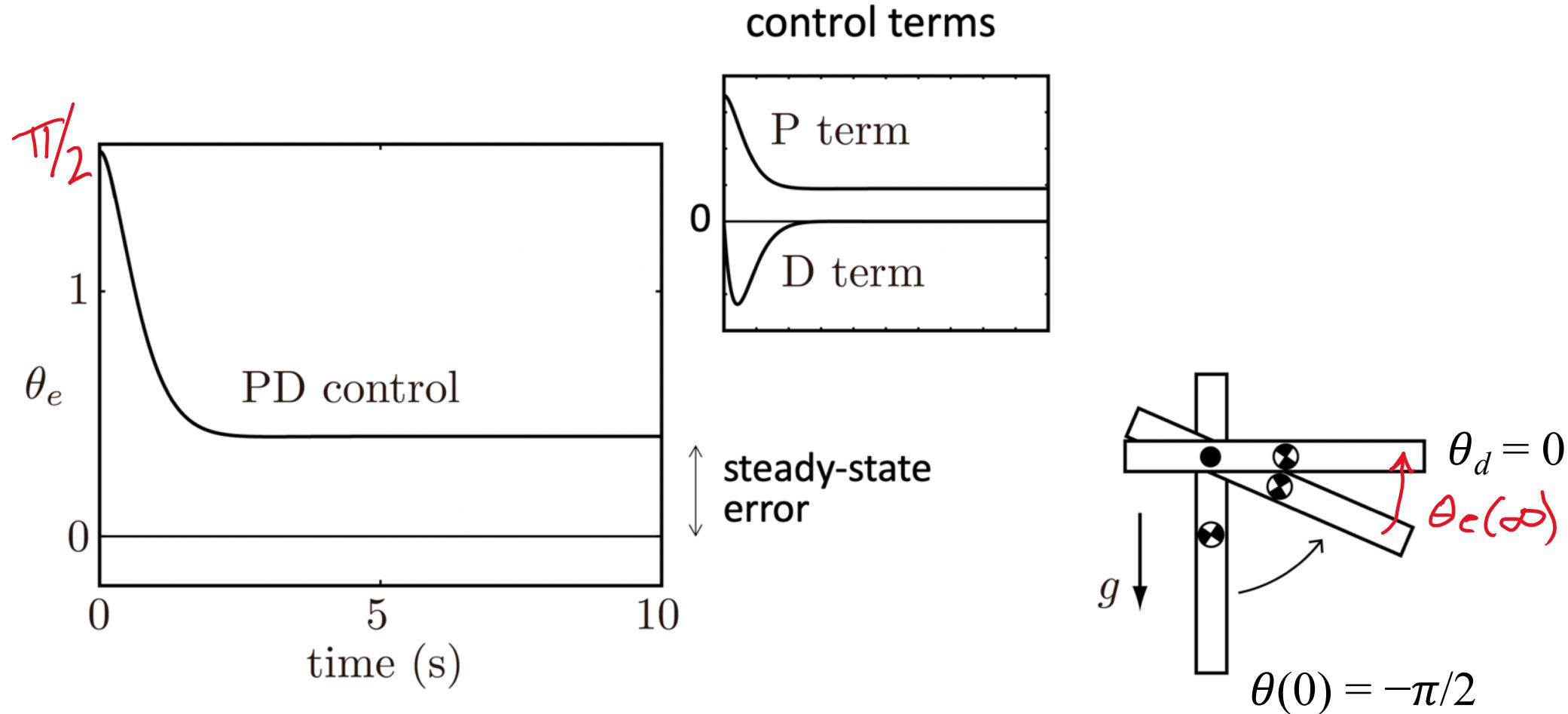
$$M \ddot{\theta}_e + (b + K_d) \dot{\theta}_e + K_p \theta_e = m g r \cos \theta$$

$$\theta_e(t \rightarrow \infty) = \frac{m g r \cos \theta}{K_p}$$

Nonhomogeneous.  
What is the steady-state error?



## Important concepts, symbols, and equations (cont.)



## Important concepts, symbols, and equations (cont.)

**Setpoint PID control**,  $g \neq 0$

$$\tau = K_p \theta_e + K_i \int \theta_e(t) dt + K_d \dot{\theta}_e$$

$$\tau = M \ddot{\theta} + m g r \cos \theta + b \dot{\theta}$$

$$\dot{\theta}_d = \ddot{\theta}_d = 0$$

$m g r \cos \theta$  is nearly constant near steady state

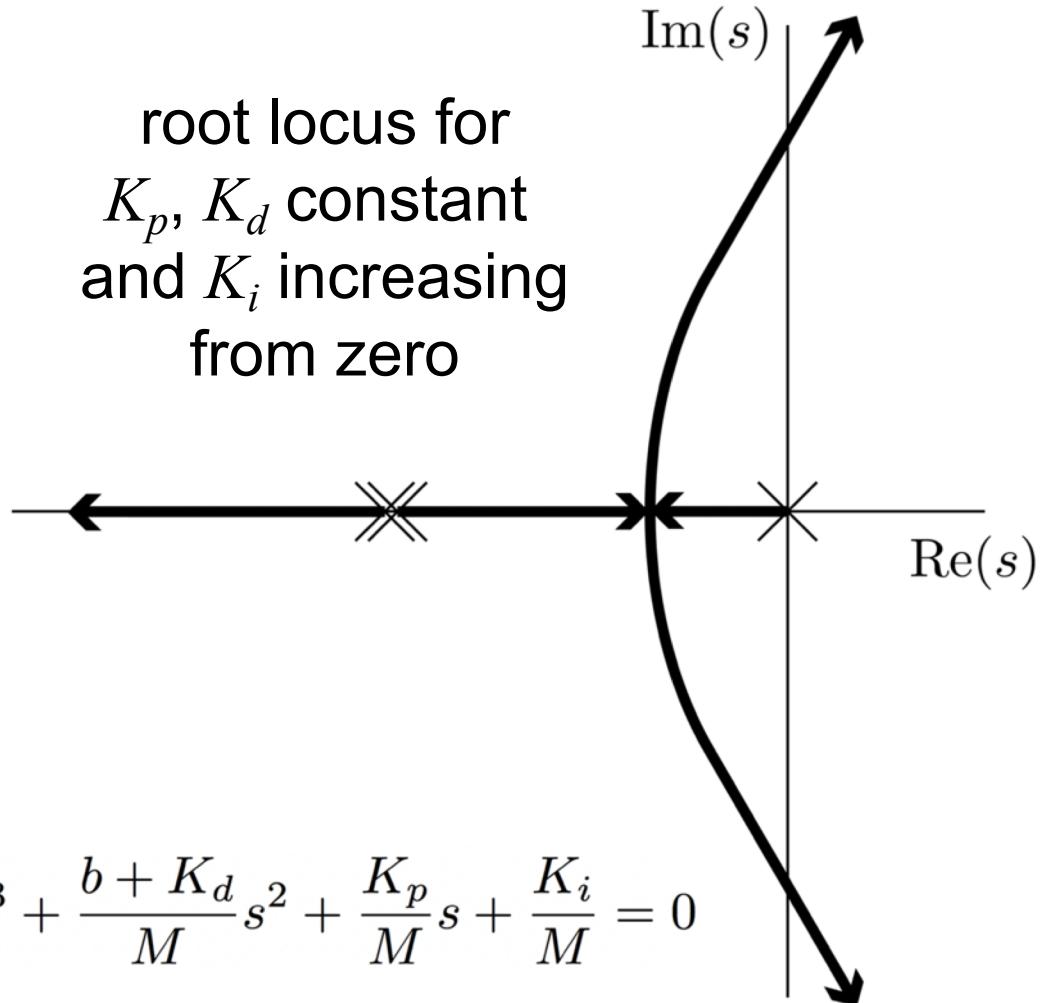
$$M \ddot{\theta}_e + (b + K_d) \dot{\theta}_e + K_p \theta_e + K_i \int \theta_e(t) dt = \tau_{\text{dist}}$$

$$M \theta_e^{(3)} + (b + K_d) \ddot{\theta}_e + K_p \dot{\theta}_e + K_i \theta_e = 0$$

$$s^3 + \frac{b + K_d}{M} s^2 + \frac{K_p}{M} s + \frac{K_i}{M} = 0$$

## Important concepts, symbols, and equations (cont.)

root locus for  
 $K_p, K_d$  constant  
and  $K_i$  increasing  
from zero

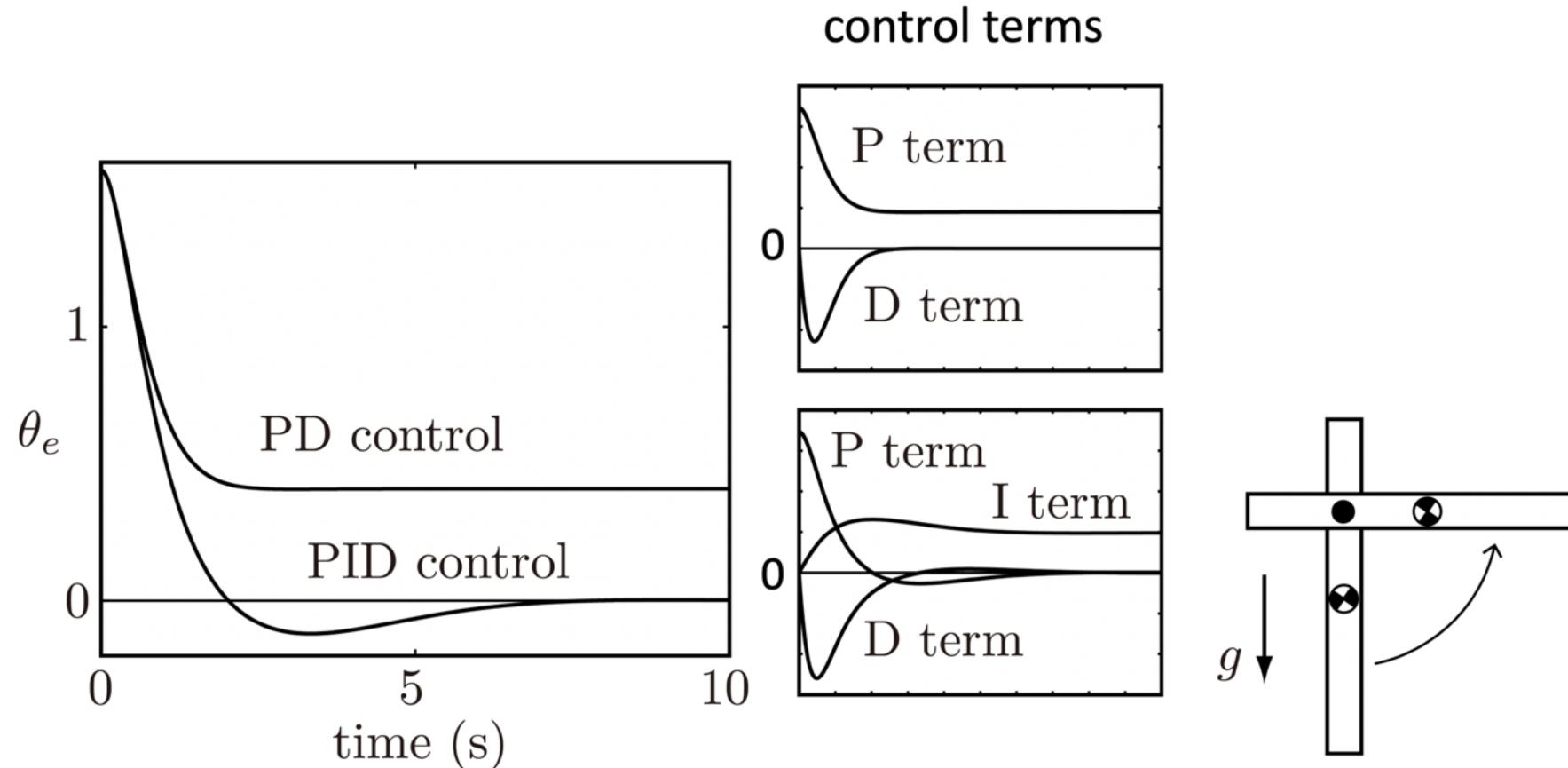


$$\begin{aligned} K_d &> -b \\ K_p &> 0 \end{aligned}$$

$$\frac{(b + K_d)K_p}{M} > K_i > 0$$

$K_i$  improves steady-state response but can worsen the transient response.

## Important concepts, symbols, and equations (cont.)



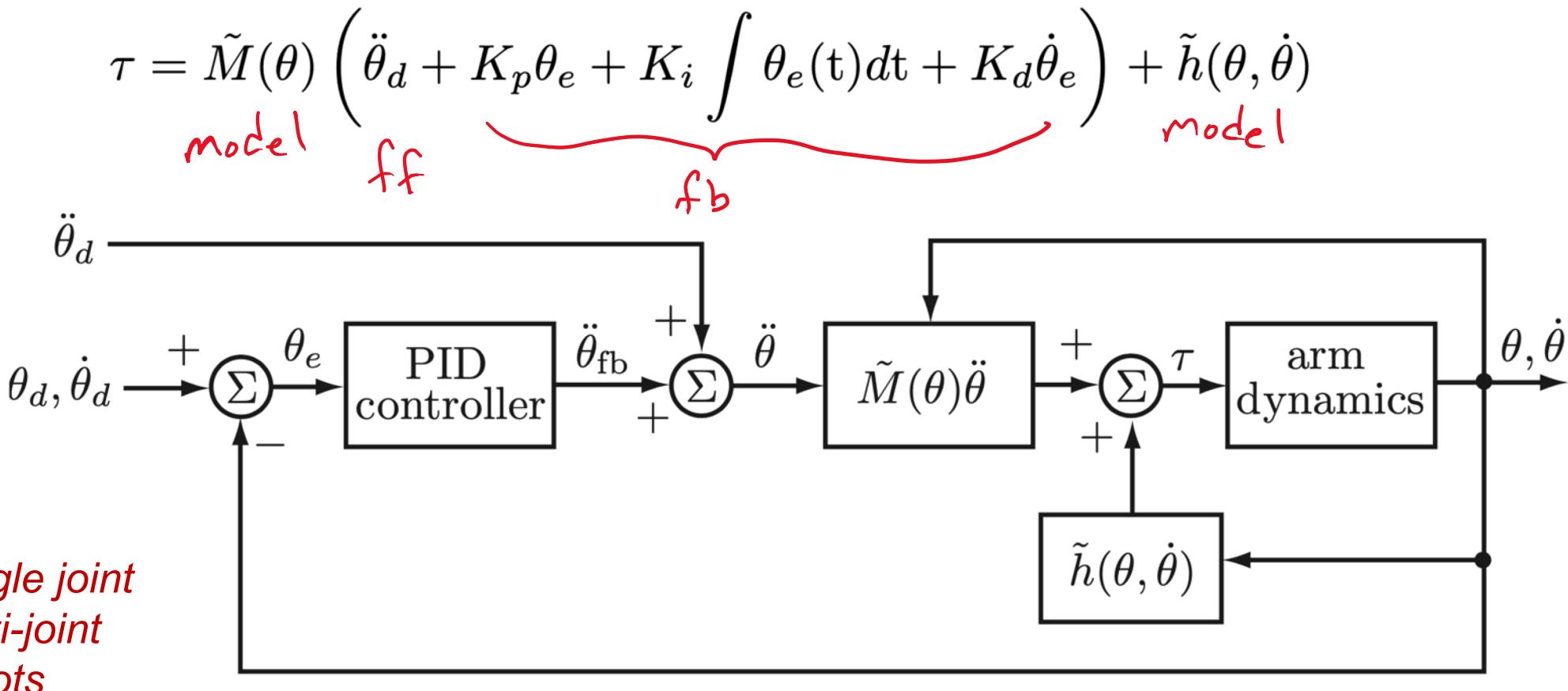
## Important concepts, symbols, and equations (cont.)

What about tracking general trajectories, not just setpoint control?

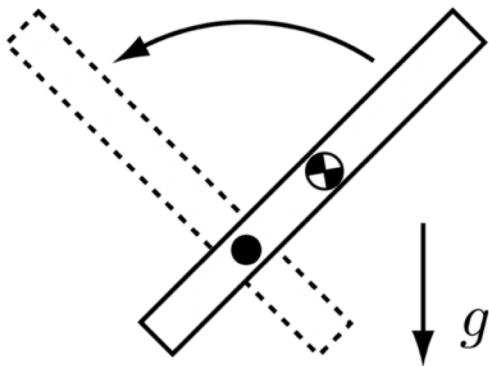
$$\tau = M \left( \underbrace{\ddot{\theta}_d + K_p \theta_e + K_i \int \theta_e dt}_{\text{ff}} + \underbrace{K_d \dot{\theta}_e}_{\text{fb}} \right) + h(\theta, \dot{\theta})$$
$$\ddot{\theta}_e = \ddot{\theta}_d - \ddot{\theta}$$
$$\ddot{\theta}_e = -K_d \dot{\theta}_e - K_p \theta_e - K_i \int \theta_e dt$$
$$\theta_e^{(3)} + K_d \ddot{\theta}_e + K_p \dot{\theta}_e + K_i \theta_e = 0$$

## Important concepts, symbols, and equations (cont.)

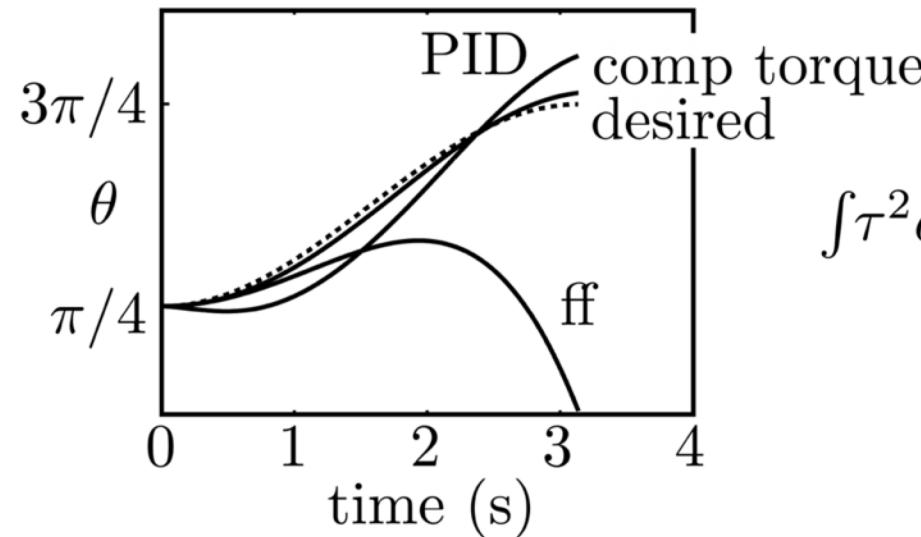
### Computed torque control (feedback linearization)



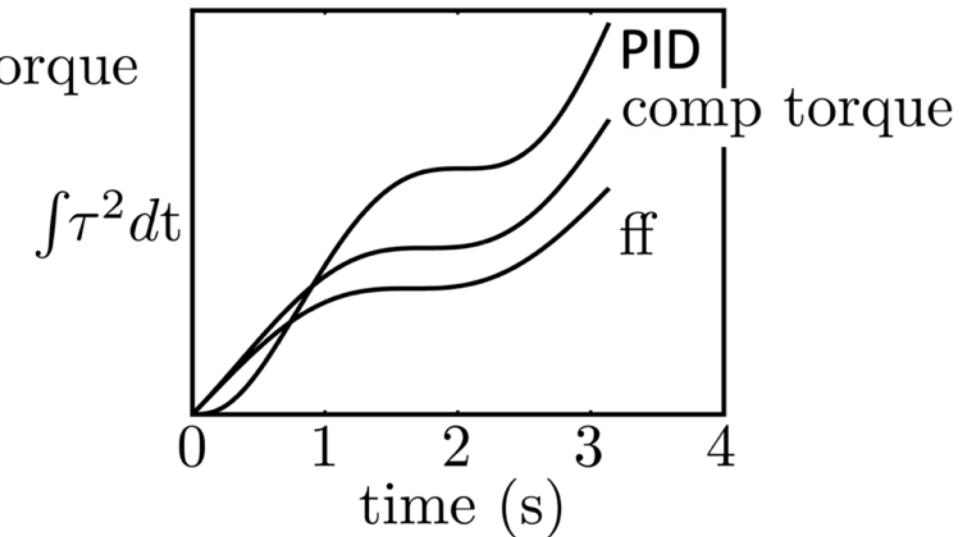
## Important concepts, symbols, and equations (cont.)



assume the  
model has some  
error



better tracking  
than ff or PID



less "effort" than PID

## Important concepts, symbols, and equations (cont.)

### Task-space computed torque control

$$\mathcal{F}_b = \Lambda(\theta) \dot{\mathcal{V}}_b + \eta(\theta, \mathcal{V}_b)$$

dynamic model:  $\{\tilde{\Lambda}, \tilde{\eta}\}$

$$\tau = \tilde{M}(\theta) \left( \ddot{\theta}_d + K_p \theta_e + K_i \int \theta_e dt + K_d \dot{\theta}_e \right) + \tilde{h}(\theta, \dot{\theta})$$

$$\mathcal{F}_b = \tilde{\Lambda}(\theta) \left( \dot{\mathcal{V}}_d + K_p X_e + K_i \int X_e dt + K_d \mathcal{V}_e \right) + \tilde{\eta}(\theta, \mathcal{V}_b)$$

$$[X_e] = \log(X^{-1} X_d)$$

$$\mathcal{V}_e = [\text{Ad}_{X^{-1} X_d}] \mathcal{V}_d - \mathcal{V}_b$$

$$\tau = J_b^T(\theta) \mathcal{F}_b$$

What if your dynamic model is poor?

comp torque is appropriate if model is "reasonable" or better, not if it's poor

The characteristic equation of the error dynamics are

$$s^5 + 2s^4 + s^3 + 2s^2 + 4s + 2 = 0$$

Write the error dynamics in the form  $\dot{x} = Ax$ . Determine if the system is stable. (Use any software you want.)

$$\dot{x}_1 = \theta_e$$

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = x_3$$

$$\dot{x}_3 = x_4$$

$$\dot{x}_4 = x_5$$

$$\dot{x}_5 = -2x_1 - 4x_2 - 2x_3 - x_4 - 2x_5$$

$$\text{eig}(A) = -1, -1, -1.26,$$

$$0.68 \pm 1.09j$$

unstable due to  
positive real component

$$A = \begin{bmatrix} 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 1 \\ -2 & -4 & -2 & -1 & -2 \end{bmatrix}$$