

EN 410

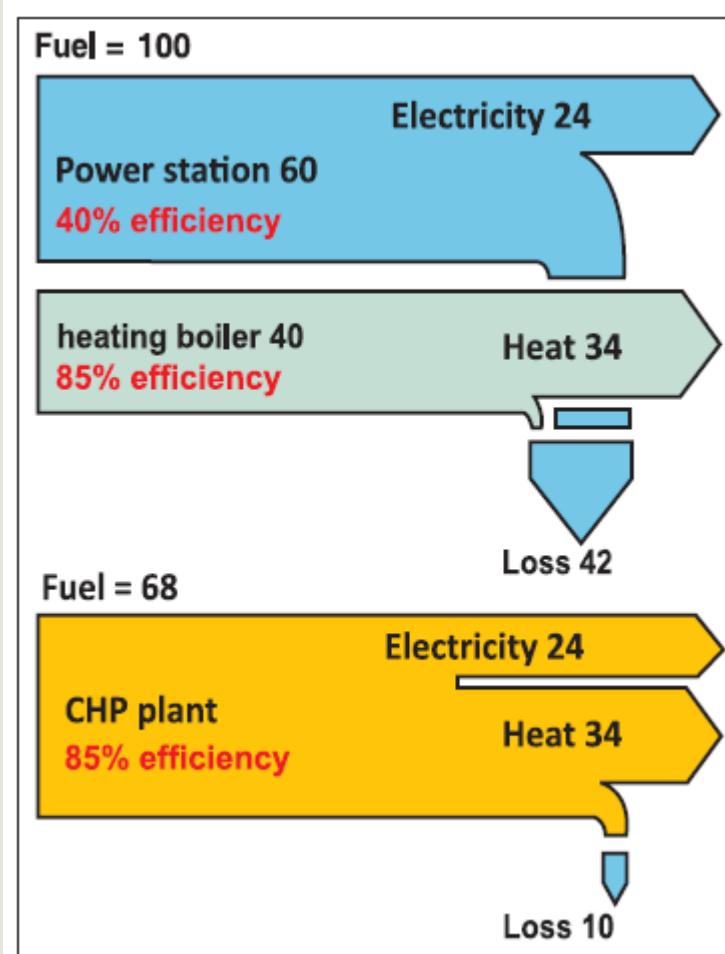
Energy Management

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Cogeneration

- Many industries require energy input in the form of heat, called **process heat**. Process heat in these industries is usually supplied by steam
- Industries that use large amounts of process heat also consume a large amount of electric power.
- The result is a plant that produces electricity while meeting the process-heat requirements of certain industrial processes (**cogeneration plant**)
- **Cogeneration:** The production of more than one useful form of energy (such as process heat and electric power) from the same energy source.

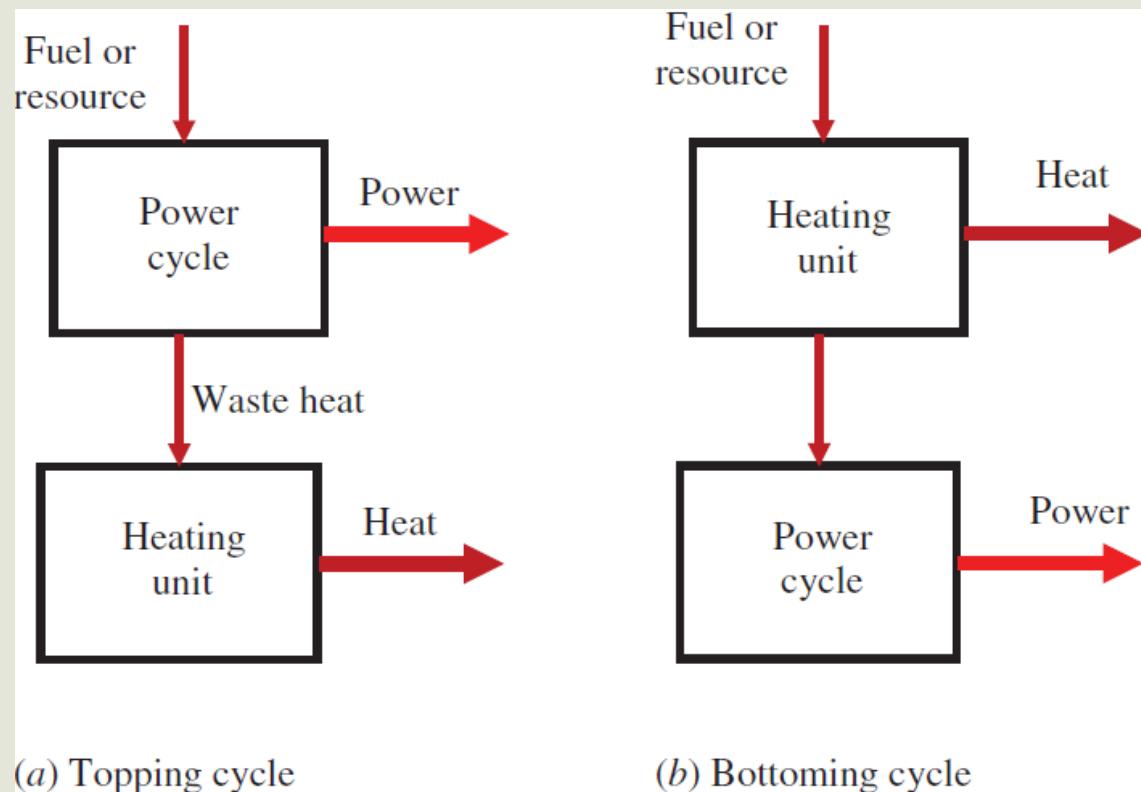
Cogeneration



- Higher efficiency – lesser fuel – lesser loss – lesser pollution
- A combination of power production and heating/cooling can also be used in a cogeneration scheme.
- It is called **trigeneration** if three useful forms of energy (such as electric power, process heat, and cooling) are produced from the same energy source.

Cogeneration Schemes

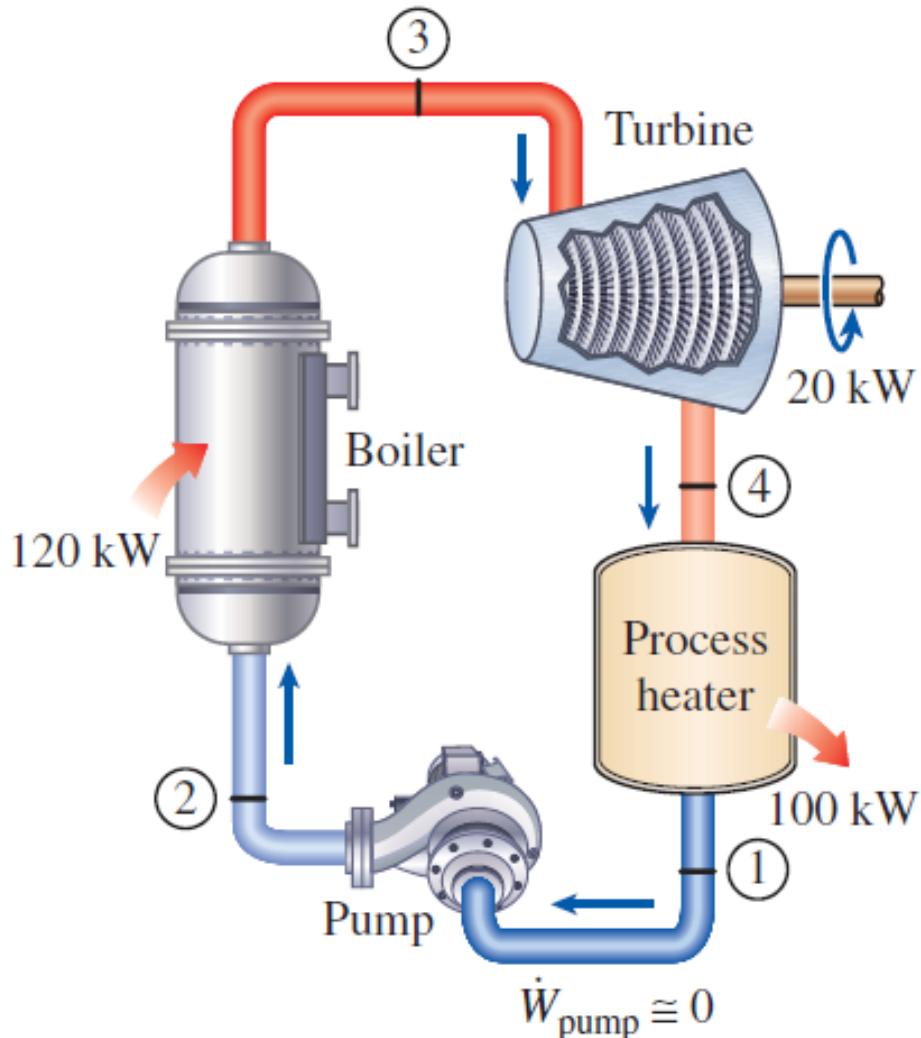
- **Topping cycle** is most commonly used. It is particularly suitable when the electric demand of the facility is greater than the thermal demand, and the process heat temperature requirements are not high.
- **Bottoming cycle** is particularly suitable when the plant needs process heat at a high temperature. Some examples include steel reheat furnaces, clay and glass kilns and aluminum remelt furnaces.
- The power cycle is also called **prime mover**.
- Steam-turbine (Rankine) cycle, a gas-turbine (Brayton) cycle, a combined cycle (combination of Rankine and Brayton cycles), an internal combustion engine



(a) Topping cycle

(b) Bottoming cycle

Steam-turbine cogeneration



Utilization factor

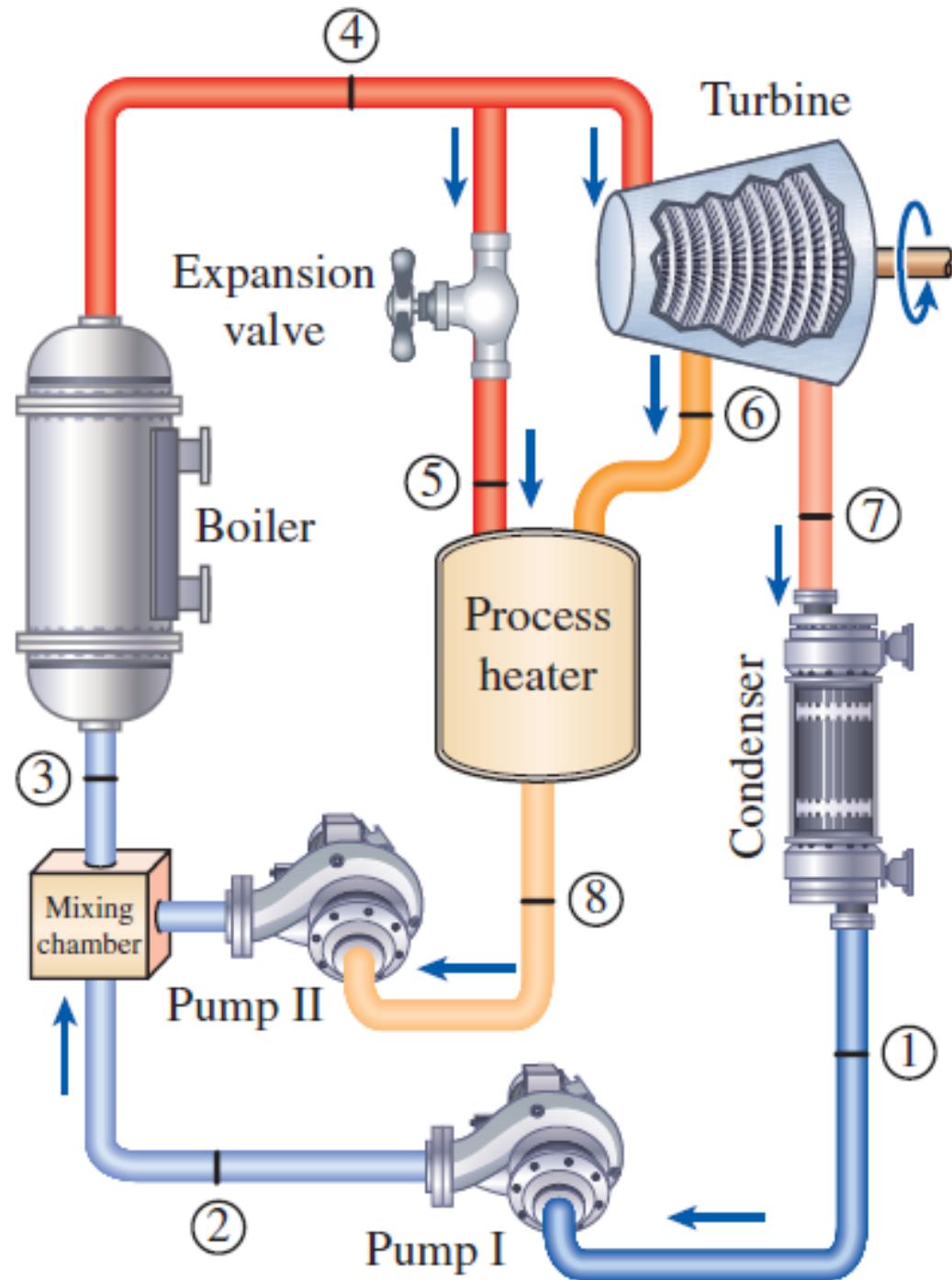
$$\epsilon_u = \frac{\text{Net power output} + \text{Process heat delivered}}{\text{Total heat input}} = \frac{\dot{W}_{\text{net}} + \dot{Q}_p}{\dot{Q}_{\text{in}}}$$

$$\text{Electricity-to-heat ratio} = \frac{\dot{W}_{\text{net}}}{\dot{Q}_{\text{process}}}$$

Ideal since not suitable for load variation

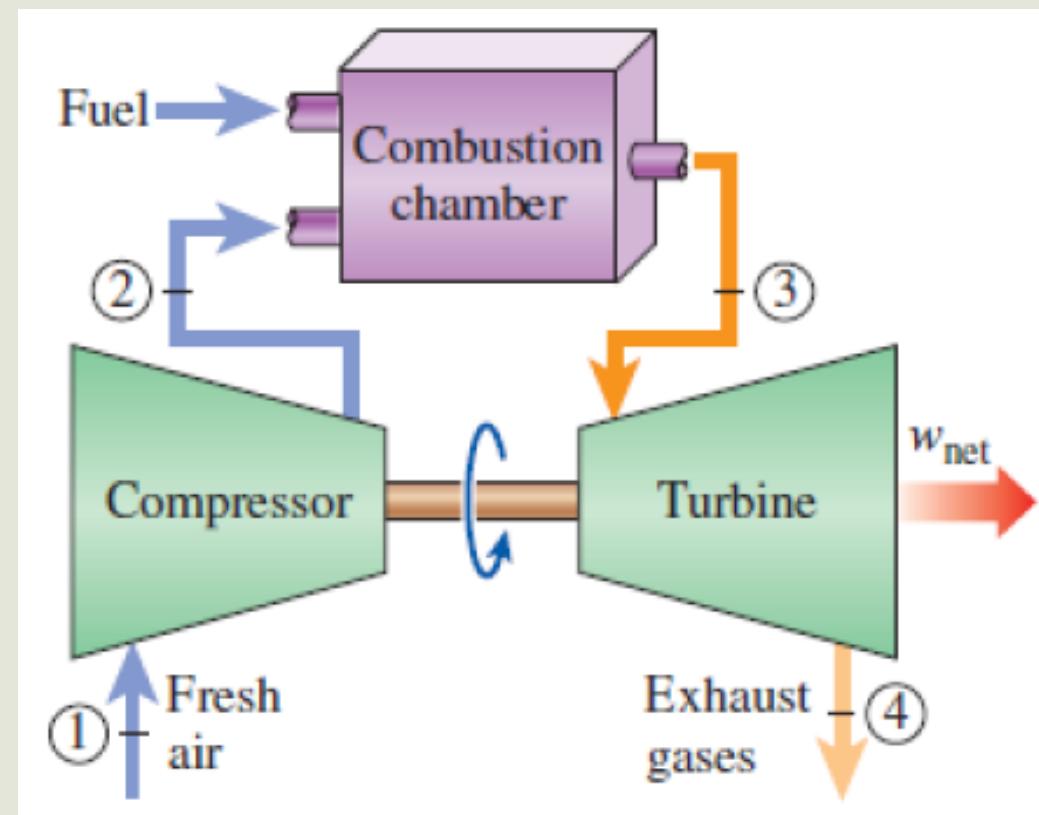
Cogeneration plant with adjustable loads

- At times of **high demand for process heat**, all the steam is routed to the process-heating units and none to the condenser ($m_7 = 0$). The waste heat is zero in this mode.
- If this is not sufficient, some steam leaving the boiler is throttled by an expansion or pressure-reducing valve and directed to the process-heating unit.
- **Maximum process heating** is realized when all the steam leaving the boiler passes through the valve ($m_5 = m_4$). No power is produced in this mode.
- When there is **no demand for process heat**, all the steam passes through the turbine and the condenser ($m_5 = m_6 = 0$), and the cogeneration plant operates as an ordinary steam power plant.



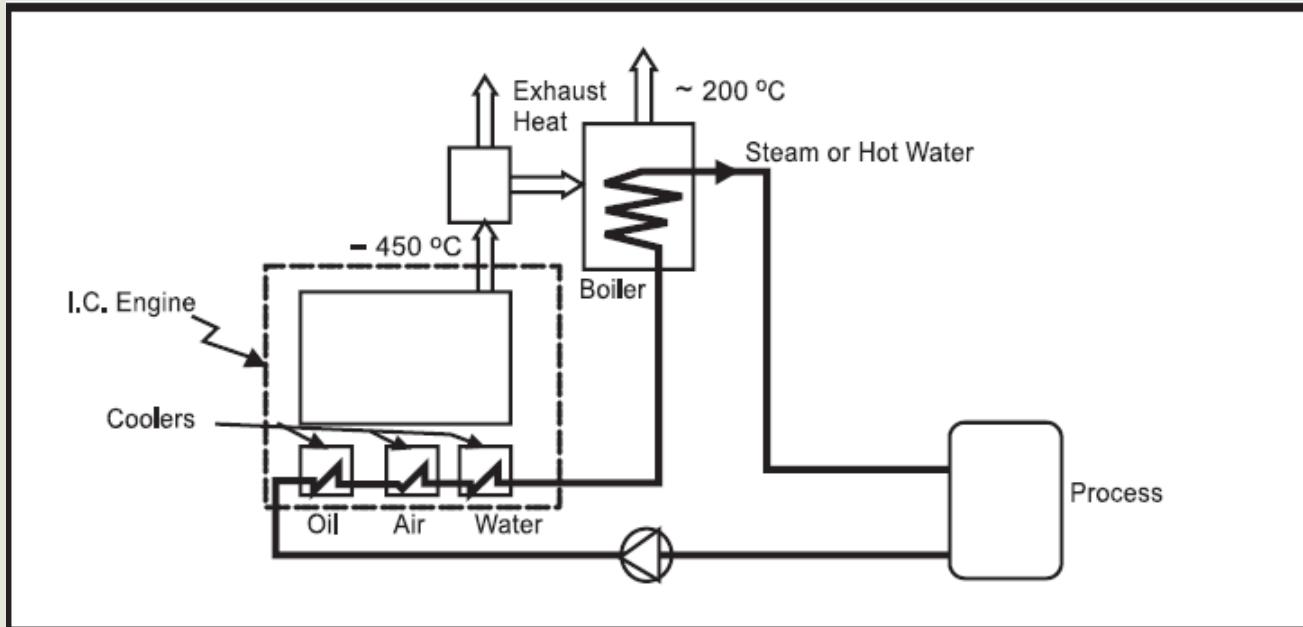
Gas-turbine cogeneration

- Heat is rejected to the atmosphere at a high temperature (300 to 400°C) which makes this plant very suitable for cogeneration applications.
- Steam is usually taken at an intermediate stage in the turbine or from the outlet of the boiler so that the temperature is sufficiently high for process heating.
- Multiple process heating at high temperature
- **Advantages :** Simpler, easier and quicker to install and operate, and cheaper compared to steam power based cogeneration.



Diesel-engine cogeneration

- **Fuel:** relatively inexpensive heavy fuel oil, a low-grade product of oil refineries. Natural gas is also used.
- **Set up time:** Diesel engine power plants can be set up quickly, normally in less than 12 months, to generate hundreds of megawatts of power.
- **Power capacity:** Reciprocation engines are mostly employed in low and medium power generation units. 50 kW to 10 MW - natural gas, 50 kW to 50 MW - light diesel fuel, 2.5 MW to 50MW - heavy fuel oil
- **Sources of waste heat:** Exhaust gas, engine jacket cooling water, lube oil cooling water, and turbocharger intercooling
- **Hot water and steam production:** Recovered heat is generally in the form of hot water or low pressure steam (about 2 bar). The high temperature exhaust gas can generate medium pressure steam (about 10 bar).



Selection of prime movers

	<i>Reciprocating engine</i>	<i>Gas turbine</i>	<i>Steam turbine</i>
<i>Application</i>	Small and medium size plants	Small, medium, and large plants	Large plants, residential and commercial districts
<i>Capacity</i>	50 – 3000 kW	500 – 30,000 kW	> 1000 kW
<i>Fuel</i>	Diesel, fuel-oil, natural gas, propane	Natural gas, diesel	Coal, natural gas, Diesel, fuel-oil, propane, biofuel, pellet
<i>Thermal efficiency</i>	Approaching 50%	Around 30%	30 – 45% Combined cycle: 50 – 60%
<i>Maintenance requirements</i>	High	Low	Low

Load Factor

Electricity and heat use: A preliminary investigation of a proposed cogeneration application should start with the analysis of plant's energy use profile. The upper and lower limits of electricity and heat use of the facility should be analyzed.

$$\text{Average electric demand (kW)} = \frac{\text{Total electricity consumption (kWh)}}{\text{Annual operating hours (h)}}$$

$$\text{Electric load factor} = \frac{\text{Average electric demand (kW)}}{\text{Peak electric demand (kW)}}$$

$$\text{Average thermal demand (kJ/h)} = \frac{\eta_{\text{boiler}} \times \text{Total fuel consumption (kg)} \times \text{HHV (kJ/kg)}}{\text{Annual operating hours (h)}}$$

$$\text{Thermal load factor} = \frac{\text{Average thermal demand (kJ/h)}}{\text{Peak thermal demand (kJ/h)}}$$

Thermal/electric load ratio

$$\text{Thermal/electric load ratio} = \frac{\text{Thermal demand (kW)}}{\text{Electric demand (kW)}}$$

1 mbh = 1000 Btu/h

Thermal/electric load ratio = 0.2931  Thermal/electric load ratio (mbh/kW)

A high value of thermal/electric load ratio corresponds to a higher utilization factor for a given cogeneration system.

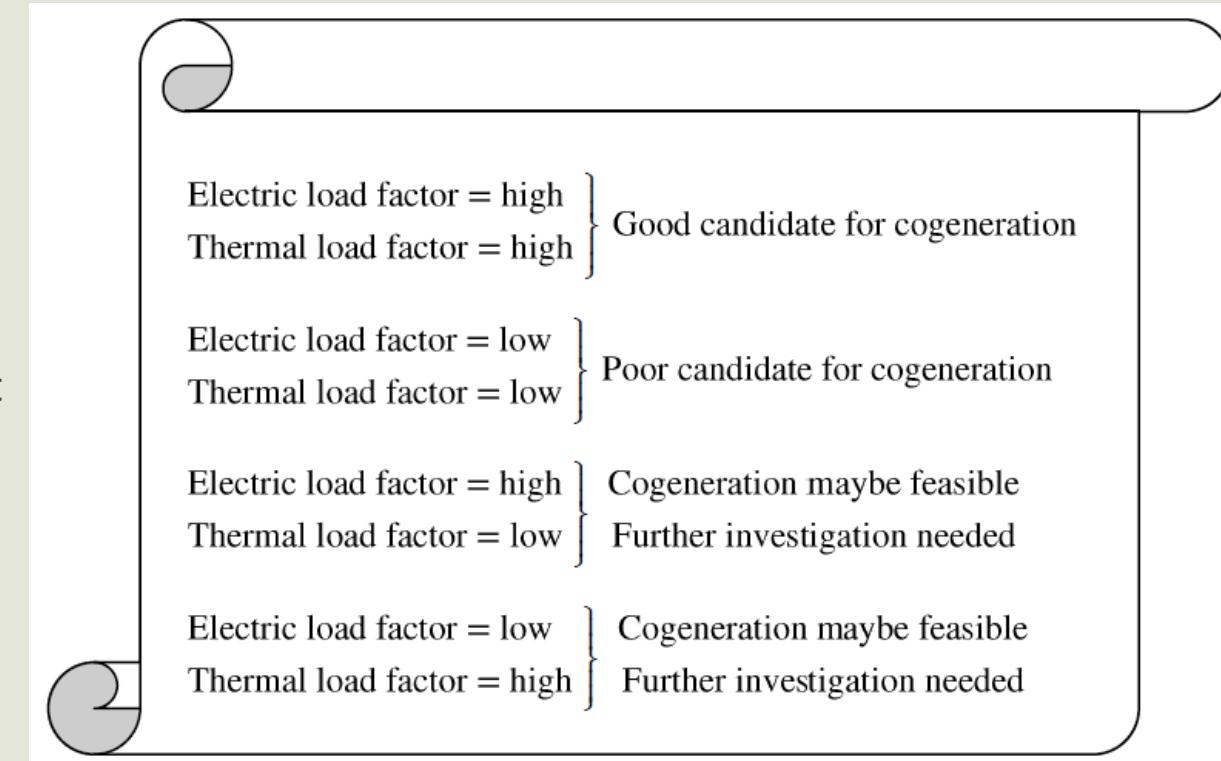
Thermal/electric load ratio > 10 mbh/kW } Excellent candidate
Thermal/electric load ratio > 3 } for cogeneration

Thermal/electric load ratio > 5 mbh/kW } Good candidate
Thermal/electric load ratio > 1.5 } for cogeneration

Thermal/electric load ratio < 2 mbh/kW } Poor candidate
Thermal/electric load ratio < 0.6 } for cogeneration

Load Factor

- A high load factor is preferred for cogeneration applications as it represents the greater use of installed capacity.
- A low load factor (e.g. 40%) means that if the cogeneration electricity capacity is selected based on the peak demand, the system will supply on average 40% of the installed capacity most of the time and close to full capacity for only a short period of time over a year.



Load Factor

- **If the electric load factor is high and the thermal load factor is low:**
- A cogeneration application can still be feasible provided that additional heat demand can be supplied from a **backup boiler**.
- A plant may have a low thermal load factor when the boiler shuts down during a certain season (summer, no space heating)
- **If the electric load factor is low and the thermal load factor is high:**
- A cogeneration application can still be feasible provided that additional electric demand can be purchased from the **utility** at a reasonable price.
- In this case, the voltage transformation and associated equipment cost may have to be taken care of by the plant.

Sizing

Electric load factor = high } Size cogeneration system
Thermal load factor = low } based on peak electric load

Electric load factor = low } Size cogeneration system
Thermal load factor = high } based on peak thermal load

➤ Thermal dispatch is more economical than electrical dispatch

➤ Electric dispatch

➤ Thermal dispatch

When both electric and heat load factors are high, the sizing of the system should be based on

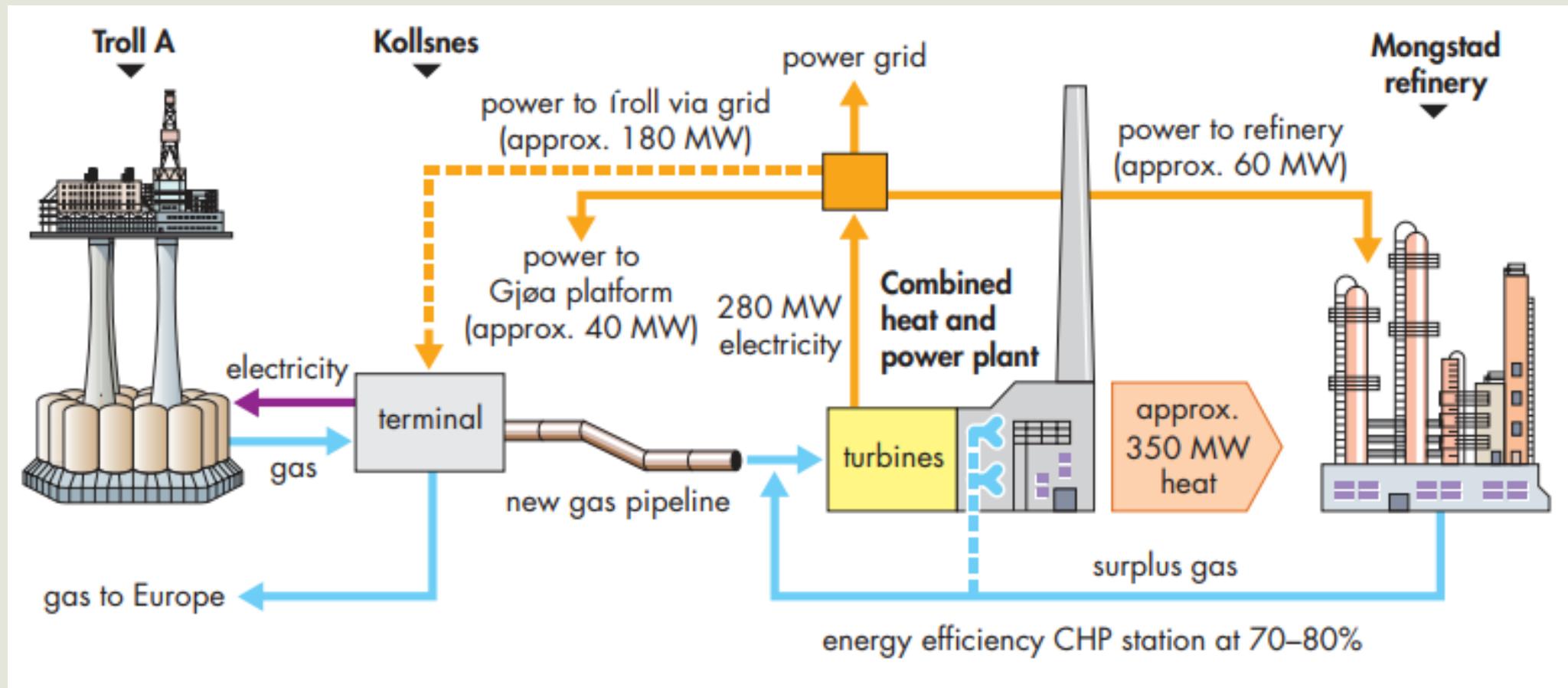
- *peak electric demand*

if the use of a backup boiler is feasible.

- *peak thermal demand*

if the selling surplus electricity to utility is feasible.

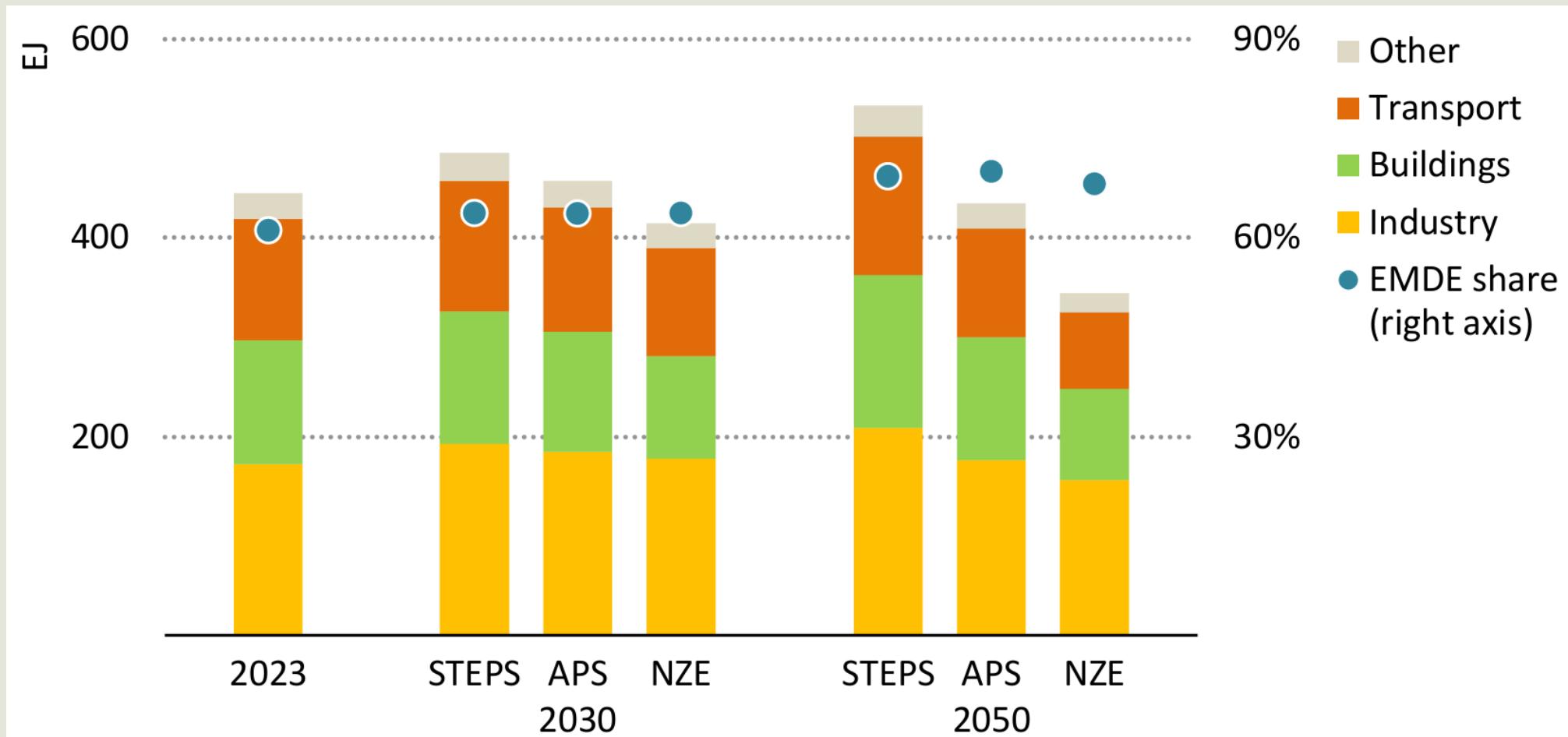
Case Study: Mongstad refinery energy project, Norway



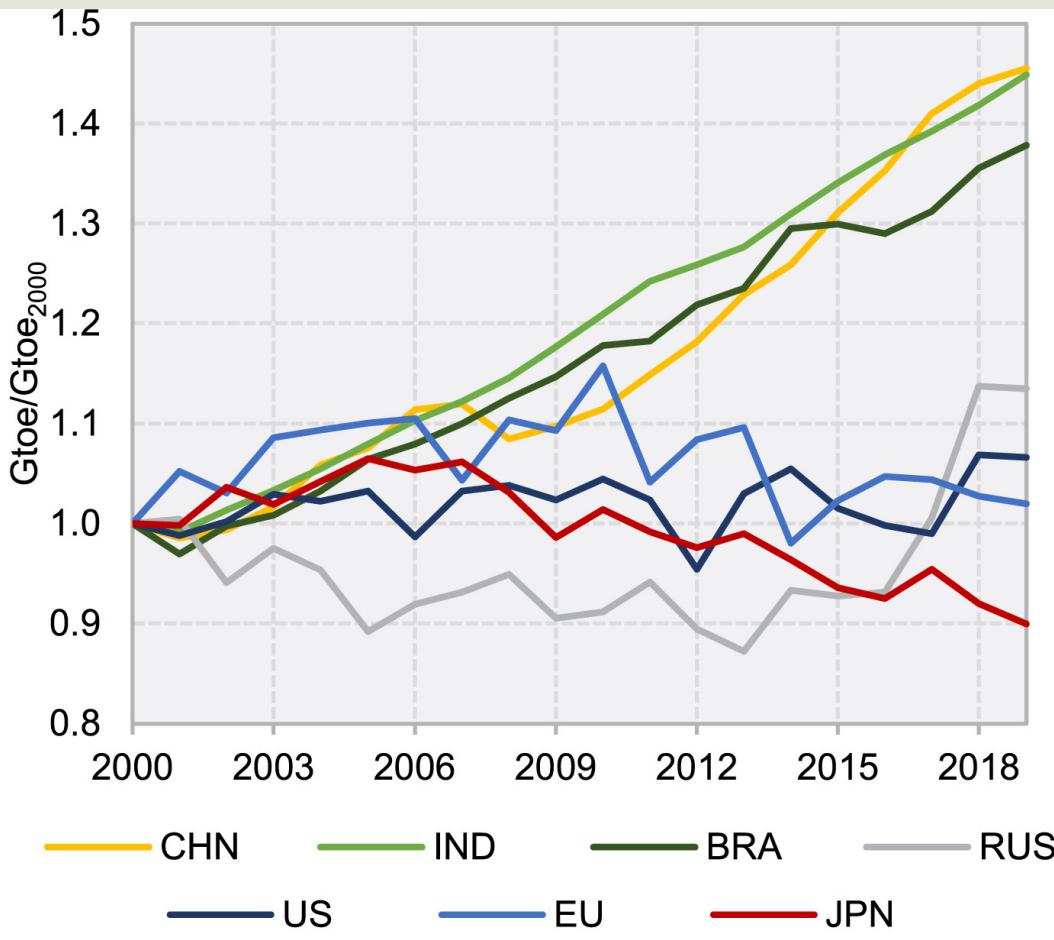
Steam-turbine cogeneration

A textile plant requires 4 kg/s of saturated steam at 2 MPa, which is extracted from the turbine of a cogeneration plant. Steam enters the turbine at 8 MPa and 500°C at a rate of 11 kg/s and leaves at 20 kPa. The extracted steam leaves the process heater as a saturated liquid and mixes with the feedwater at constant pressure. The mixture is pumped to the boiler pressure. Assuming an isentropic efficiency of 88% for the turbine, determine (a) the rate of process heat supply, (b) the net power output, and (c) the utilization factor of the plant and the electricity-to-heat ratio.

Energy demand across sectors

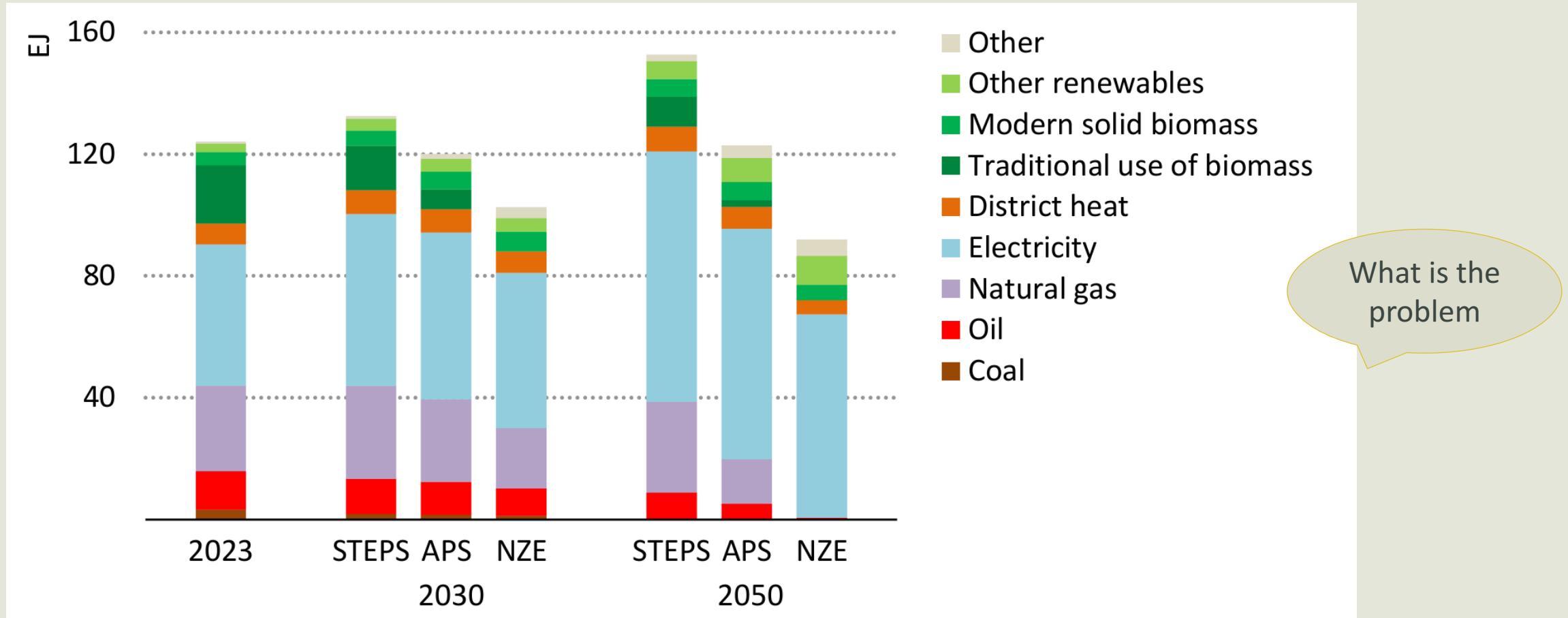


Why India?

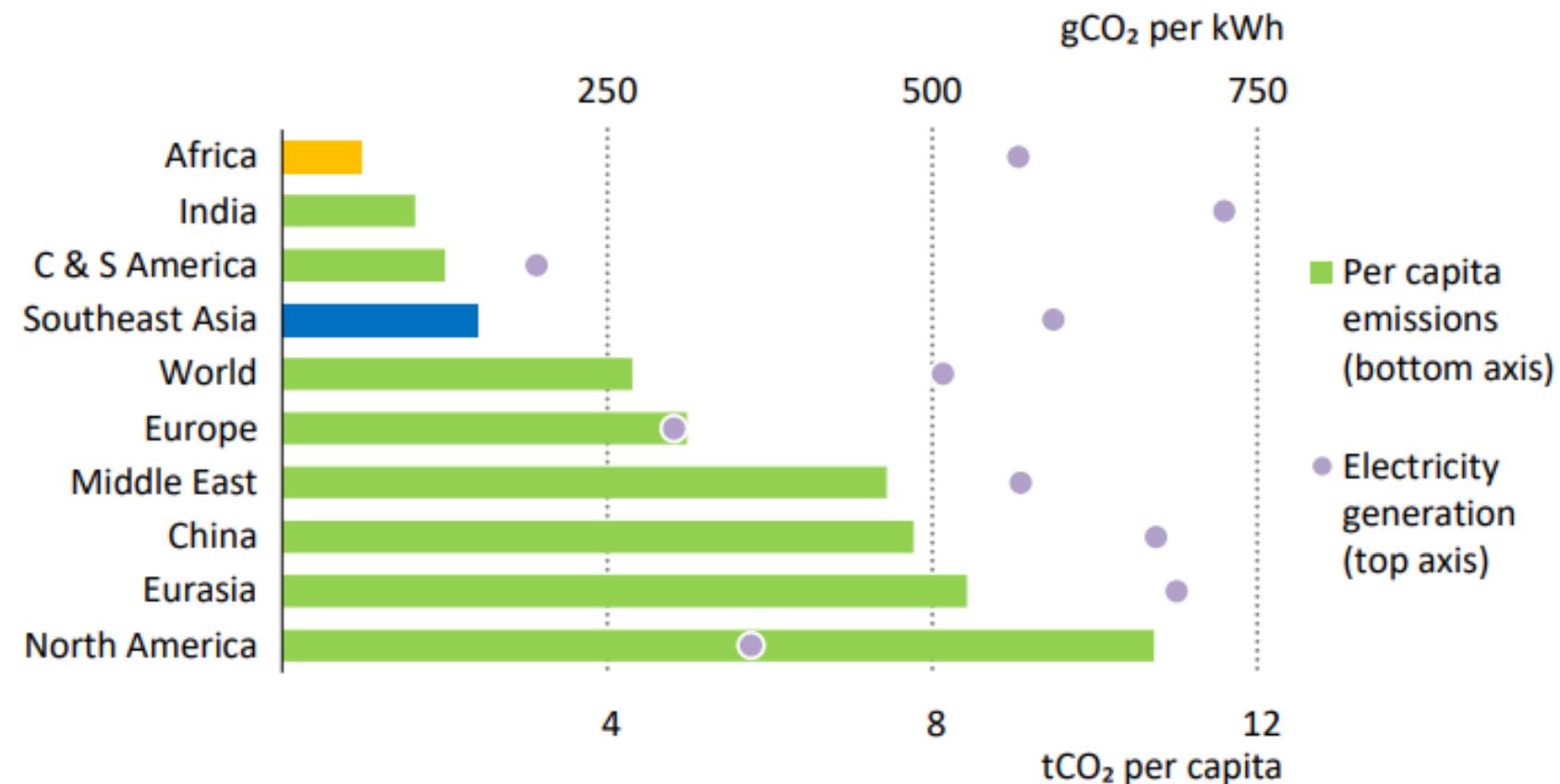


Region	2017 floor area (billion m ²)	% mandatory policy coverage, 2017	2050 floor area (billion m ²)	% growth without mandatory coverage
North America	37	83%	54	30%
European Union	29	97%	40	3%
Other advanced economies	13	96%	19	8%
China	58	100%	81	0%
India	21	9%	84	91%
Africa	21	8%	58	95%
Latin America	12	1%	26	98%
Other emerging economies	44	26%	99	84%
World	235	61%	461	71%

Building energy demand

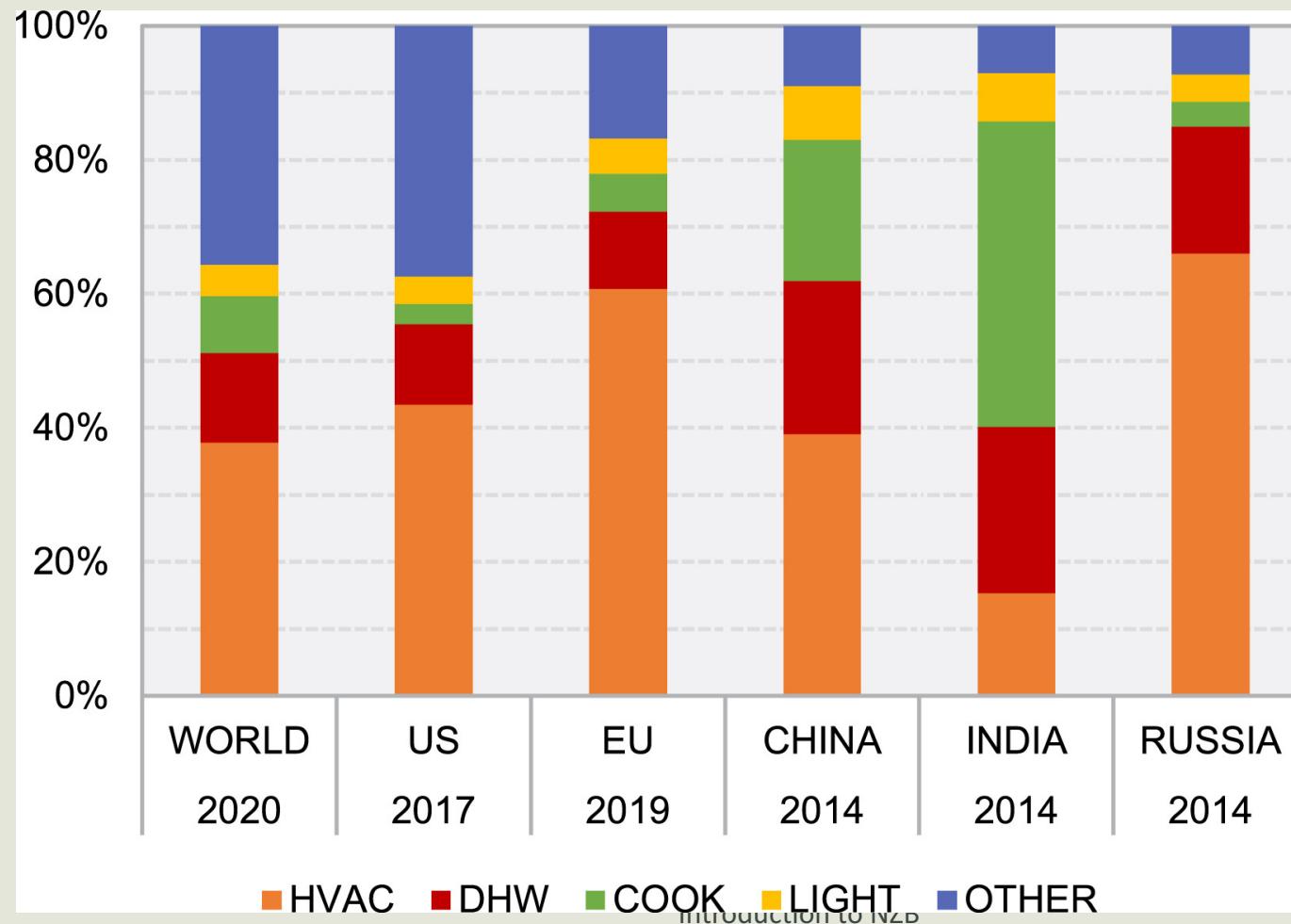


Why India?



India's per capita CO₂ emissions are 60% lower than the global average, but the emissions intensity of its electricity generation is among the highest of any country.

End use intensity global and across countries



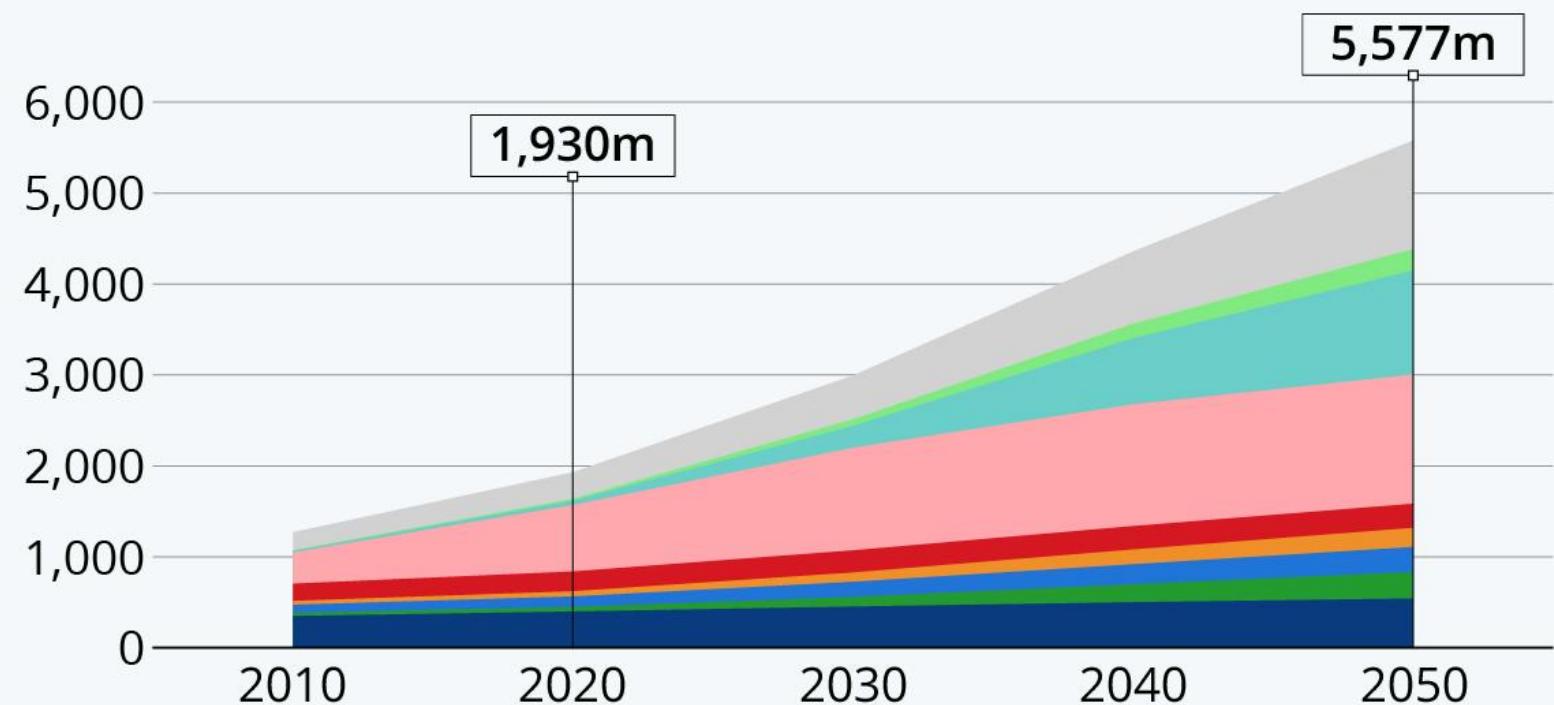
HVAC systems

- World's air conditioners and fans – 10% global electricity consumption.
- Energy demand for space cooling has risen at an average pace of 4% per year since 2000, twice as quickly as for lighting or water heating
- Higher energy consumption for space cooling particularly affects peak electricity demand, especially during hot days when equipment is used at full capacity.

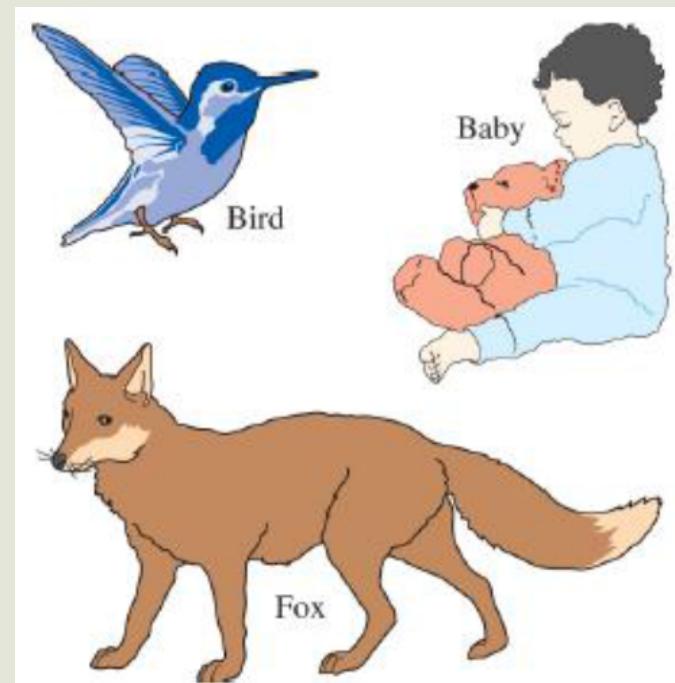
China and India Drive Global Demand for Air Conditioning

Projected number of air conditioning units in use worldwide by country/region (in millions)

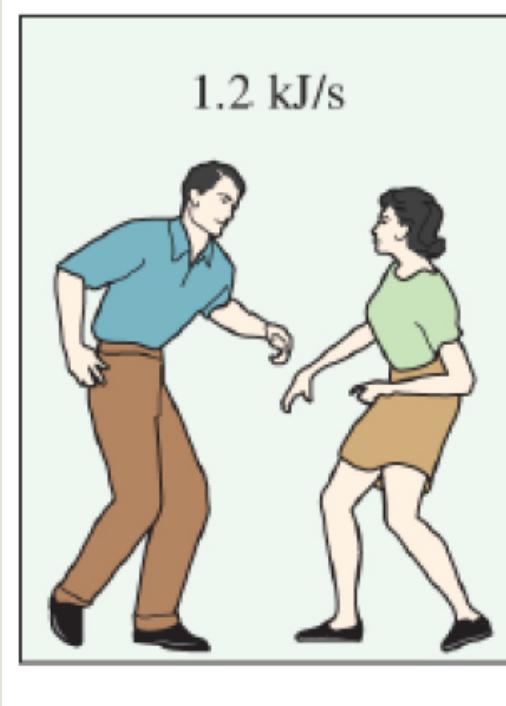
■ U.S. ■ Brazil & Mexico ■ European Union ■ Middle East
■ Japan & Korea ■ China ■ India ■ Indonesia ■ Rest of world



Why we need HVAC system?

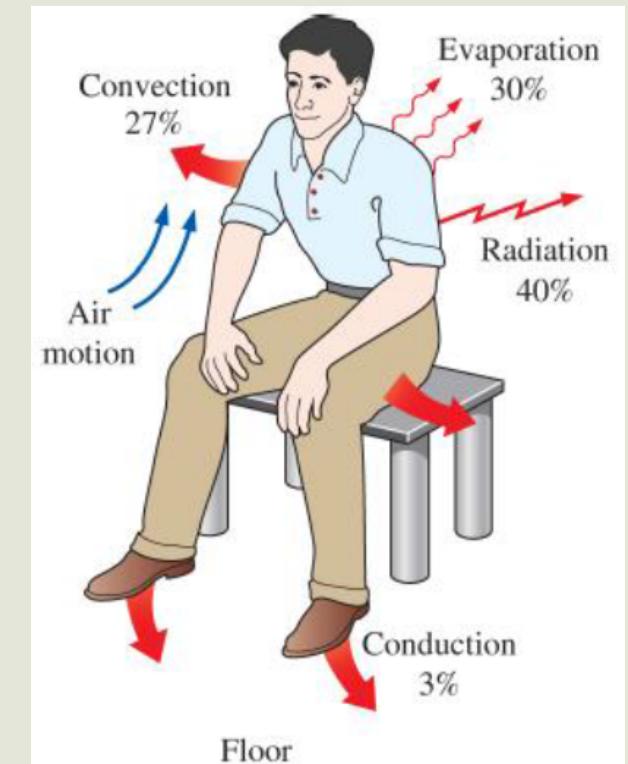
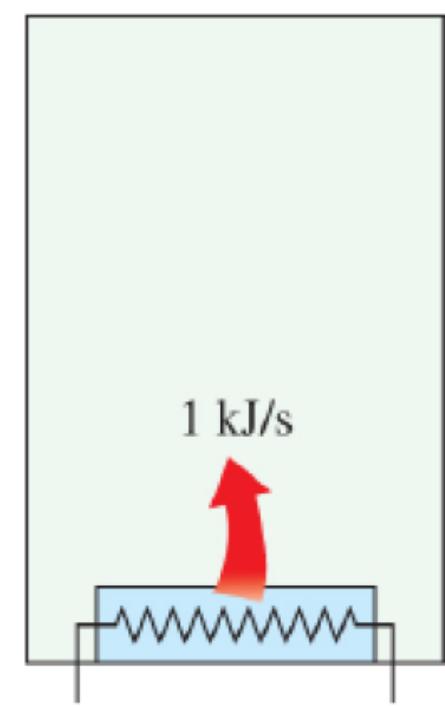


Most animals come into this world with built-in insulation, but human beings come with a delicate skin.



Two fast-dancing people supply more heat to a room than a 1-kW resistance heater.

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Mechanisms of heat loss from the human body and relative magnitudes for a resting person.

Comfort Chart

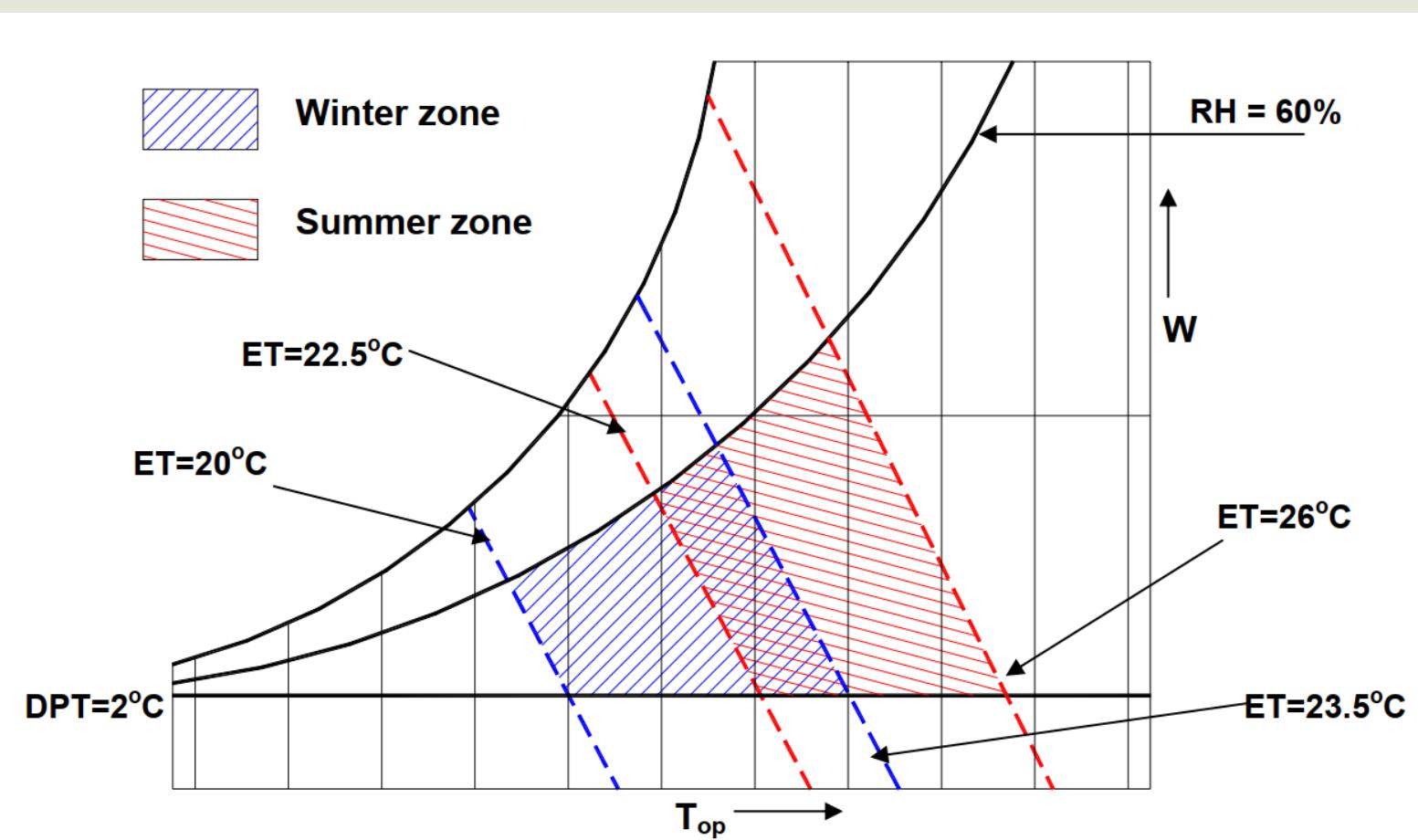
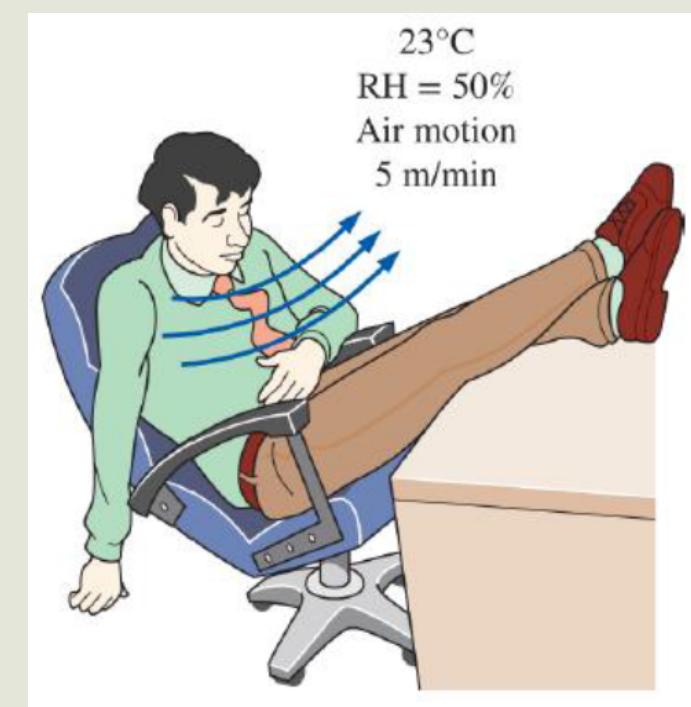
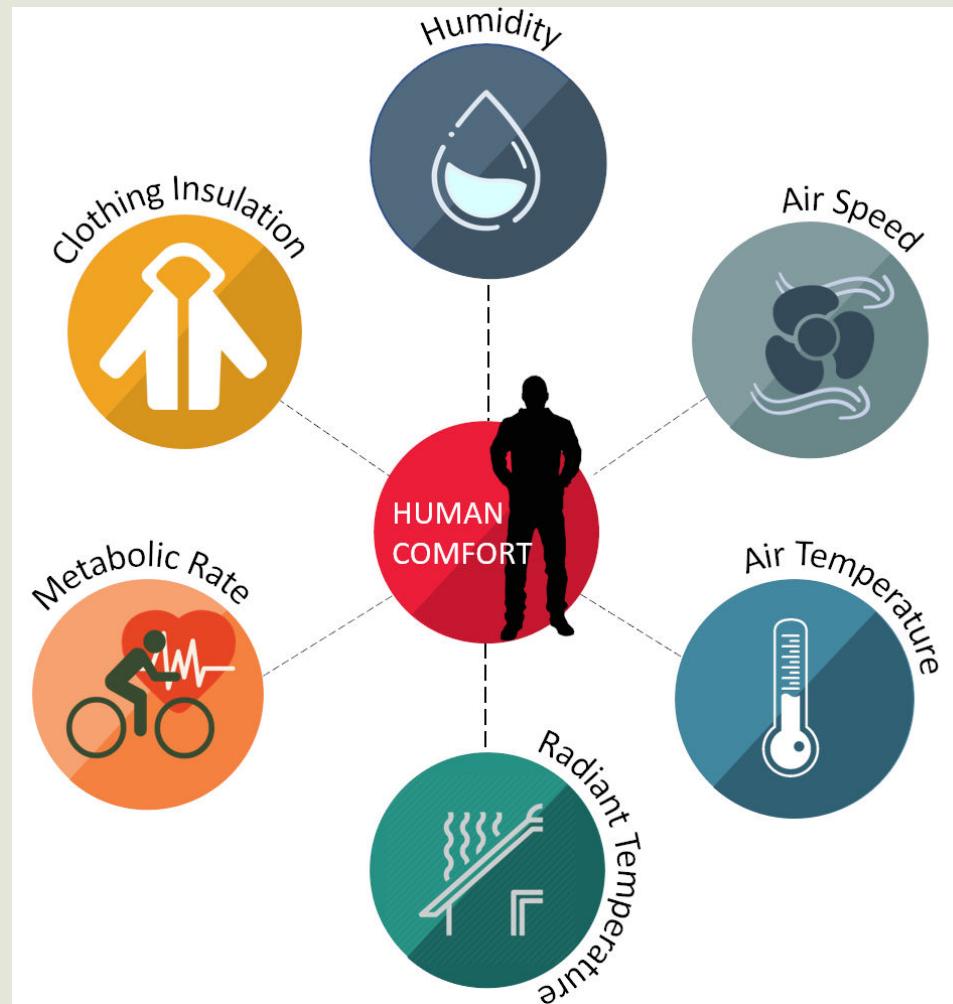


Fig.29.3: ASHRAE comfort chart for a sedentary person (activity ≈ 1.2 met)



A thermally
comfortable
environment.

Thermal comfort



Purpose of HVAC system

The purpose of comfort air conditioning is to provide an artificial environment in a given volume irrespective of ambient conditions as required for comfort and welfare of occupants by controlling dry-bulb temperature (DBT), humidity, air movement, cleanliness, noise level, etc.

Refences

- Introduction to Pinch Technology, Rokni, Masoud, DTU
- https://www.youtube.com/watch?v=MQejghnZj_w&list=PL_UhBh-E8IOLmkctKJng1EmQru3RDZyV7&index=8

