



EN 410

Energy Management

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Strategies of Net Zero Buildings - Space Cooling

- Reduce the room load
- Reduce the system load
- Meet the load by passive cooling methods

Vapor absorption system

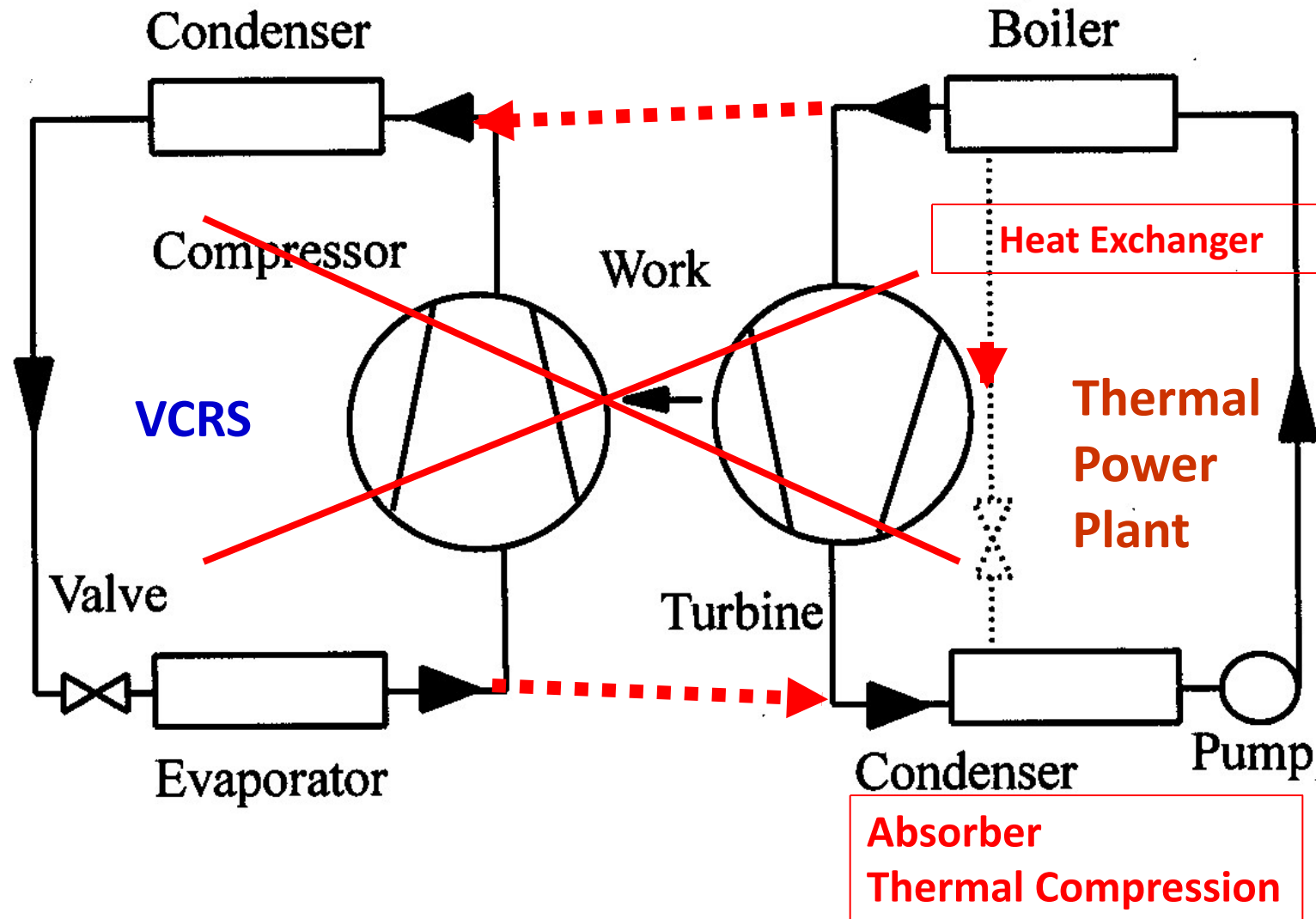
Passive cooling

Working Pairs:

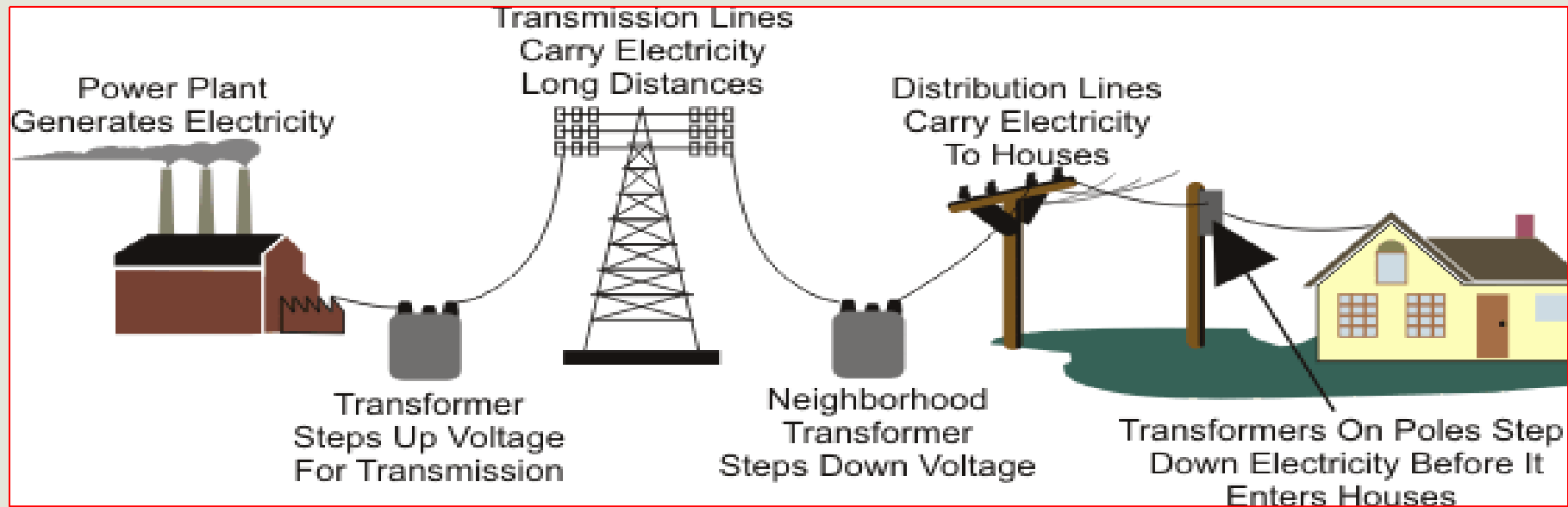
LiBr - H₂O ,

NH₃ - H₂O ,

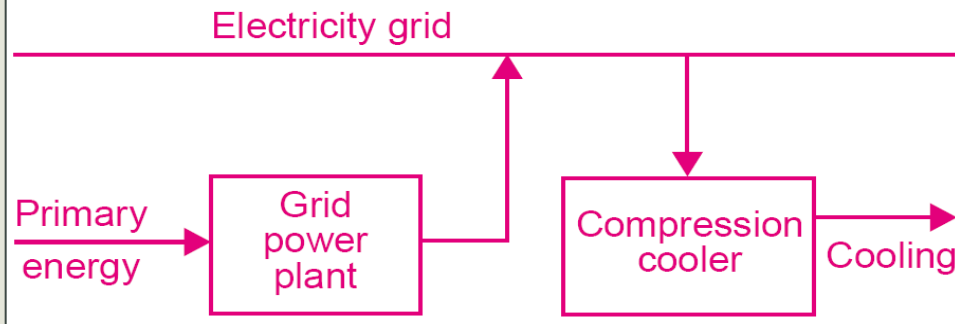
R134a - DMAC



Power generation – Transmission – Compression Cooling



Electrical compression cooling system



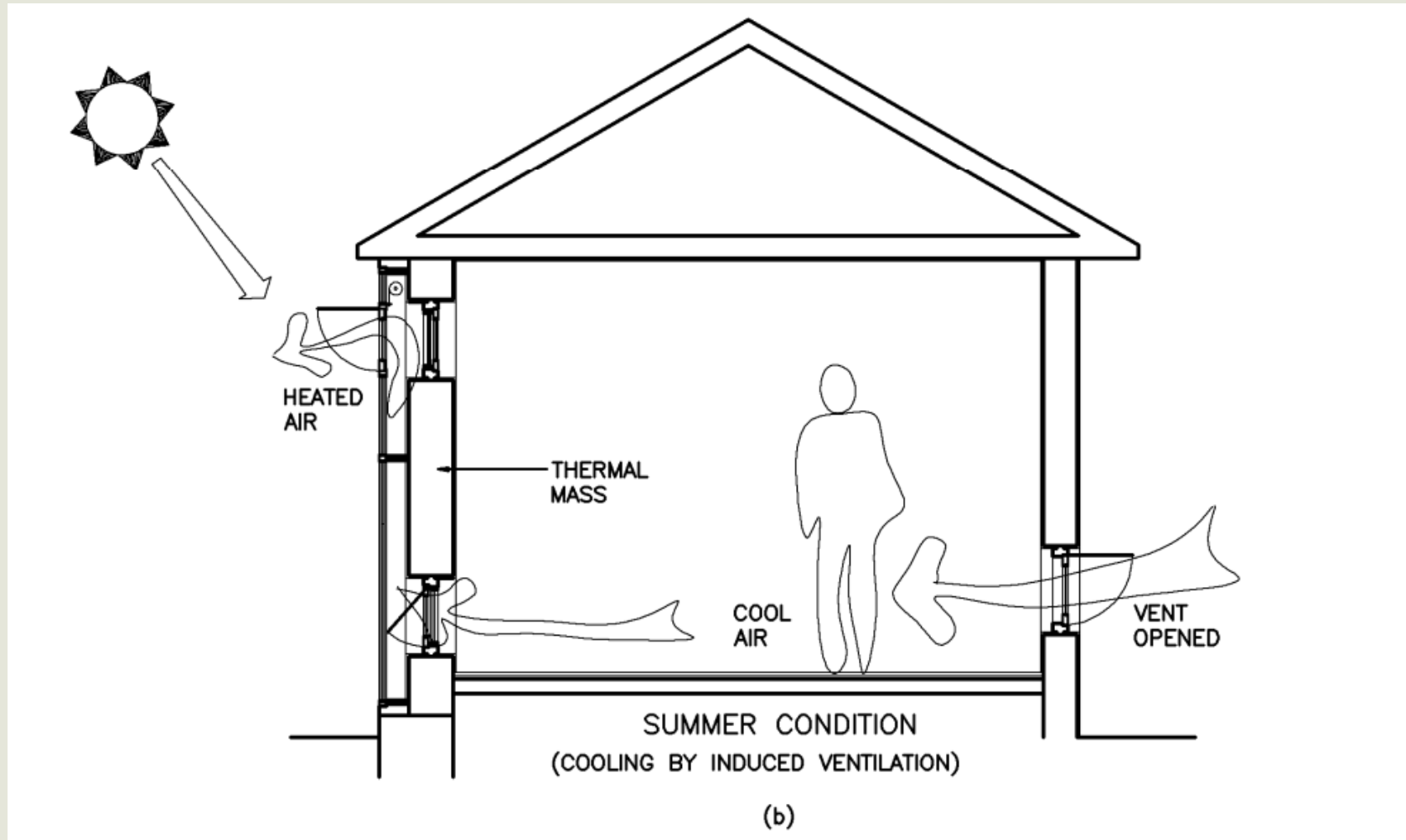
Expected losses in power lines

Power plant loss	: 70%
<i>(Efficiency = 30%)</i>	
Generator Loss	: 5%
T & D Loss	: 30%
End Equipment loss	: 10%

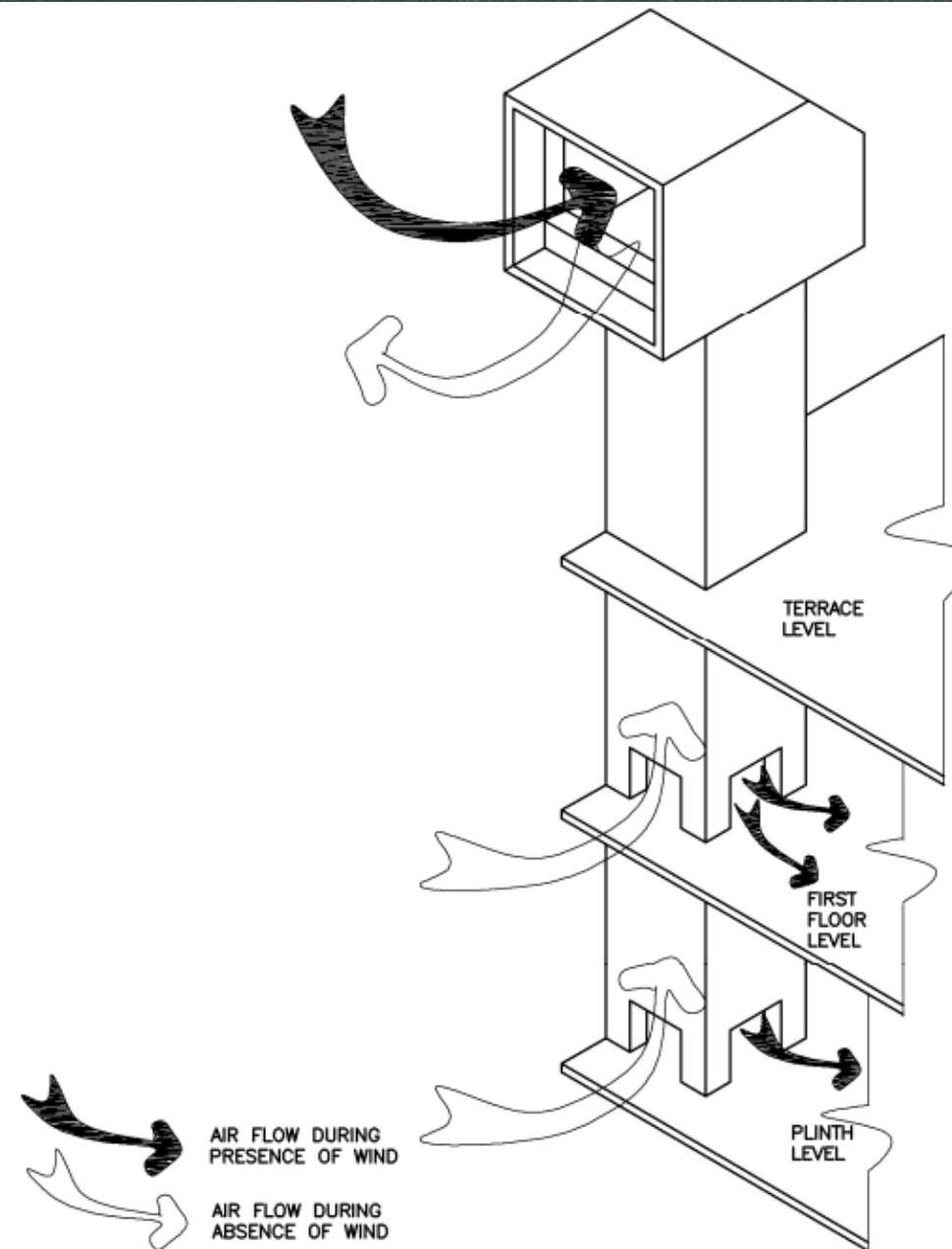
25 kW – 7.5 kW – 6.75 kW – 4.75 kW – 4.25 kW

Heat Source to Site Power Factor: $4.25/25 = 0.17$

Trombe wall / Solar chimney



Wind tower



Estimate the energy demand (EER) of an air-conditioner

Inlet condition: DBT=20°C, WBT=14°C

Outlet condition: DBT=12.7°C, WBT=11.3°C

Capacity = 20 TR

Air velocity = 2.5 m/s

Cross sectional area = 1.2 m²

Power drawn by compressor = 10.69 kW

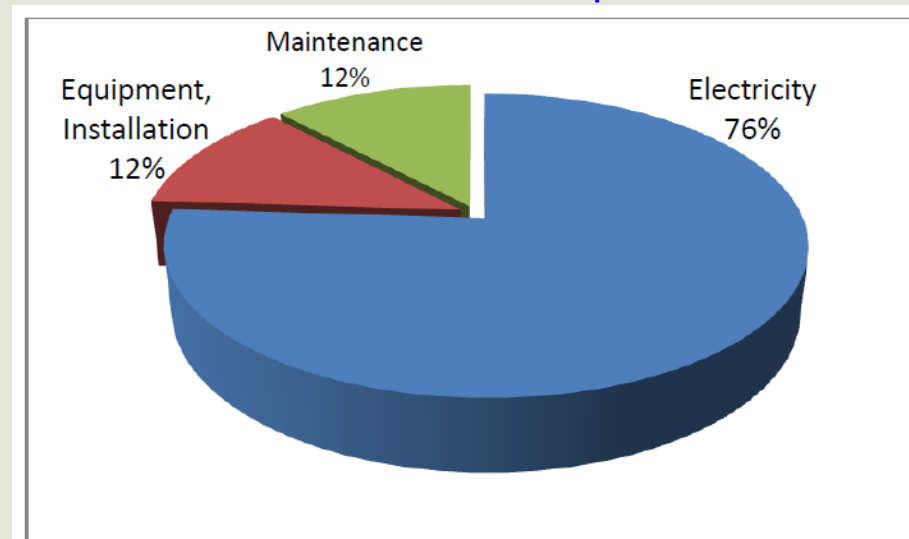
(Valid from the 1st July, 2023 to 30th June, 2025)

Indian Seasonal Energy Efficiency Ratio(kWh/kWh)		
Star Rating	Minimum	Maximum
1 Star	2.70	3.09
2 Star	3.10	3.39
3 Star	3.40	3.69
4 Star	3.70	3.99
5 Star	≥ 4.00	

Compressor

- A **compressor** is capable of compressing the gas to very high pressures.
- Commonly used in industrial facilities to perform a wide variety of tasks such as cleaning, operating pneumatic equipment, and even refrigeration.
- It is often referred to as the **fourth utility** after electricity, water, and natural gas or oil.
- *Compressors are responsible for most energy consumption in many facilities.*
- The electricity consumption associated with compressors and compressed air systems may represent 70% of total consumption
- Refrigerant or Air Compressor

A typical distribution of costs associated with compressors



Classification

- Compressors are broadly classified as positive displacement (or displacement) and dynamic compressors.
- In **positive displacement compressors**, an application of shaft work decreases the volume of the fluid chamber, thus compressing it.
- In **dynamic compressors**, an application of external work allows the transfer of angular momentum to the fluid, and this result in an increase in fluid pressure.
- **Positive displacement compressor:** reciprocating and rotary.
 - **Rotary compressor:** vane, screw, roots, and liquid ring.
 - **Reciprocating compressor:** trunk, crosshead, free piston, labyrinth, and diaphragm.
- **Dynamic compressor:** radial or centrifugal, axial, and ejector.
- Classification according to **lubricant/non-lubricant, Water/air cooled, Single/Multi Cylinder, Single/Multi stage**

Reciprocating compressor

2

When the piston moves down, the valve opens and air is pulled into the chamber

1

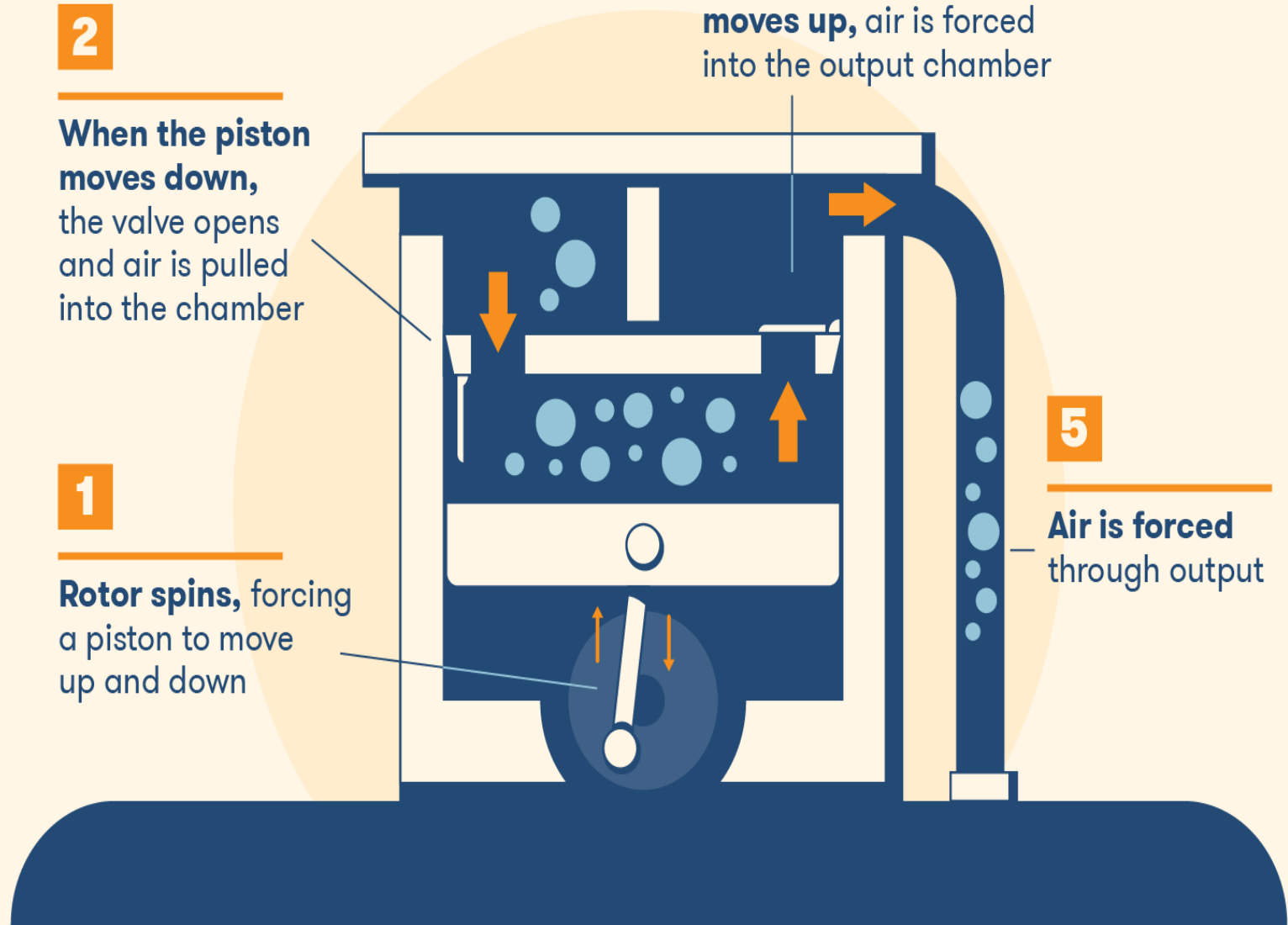
Rotor spins, forcing a piston to move up and down

3

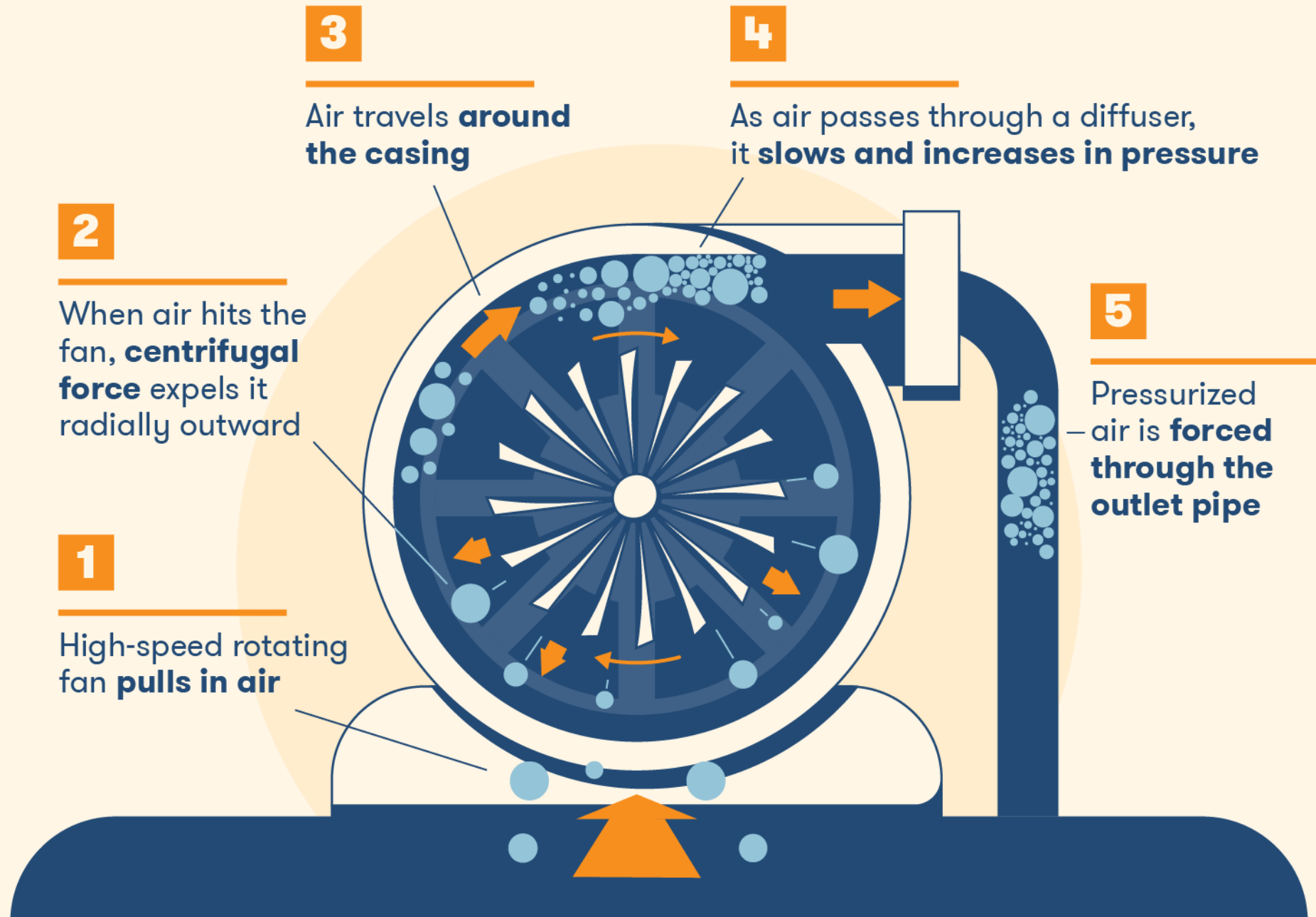
When the piston moves up, air is forced into the output chamber

5

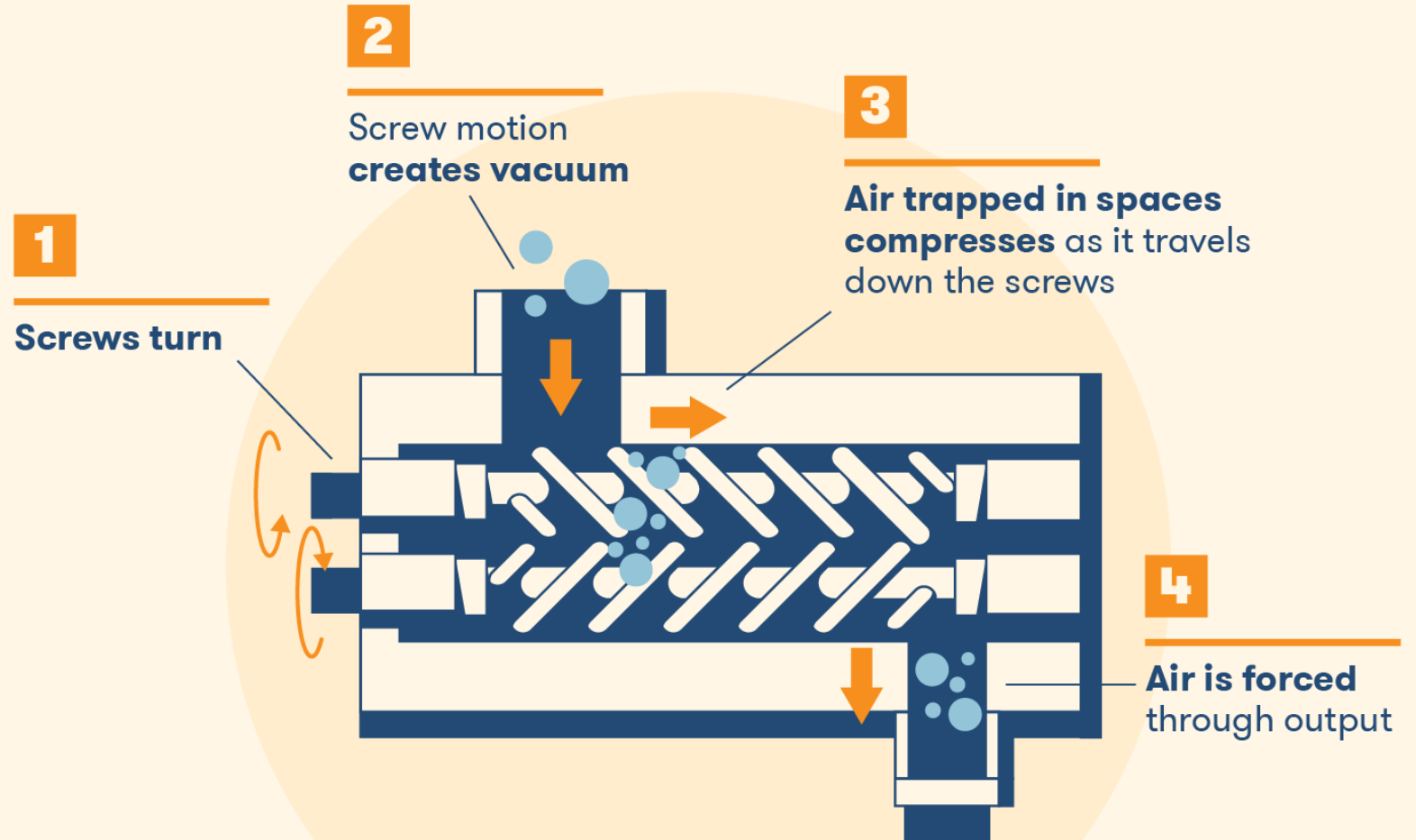
Air is forced through output



Centrifugal compressor



Screw compressor



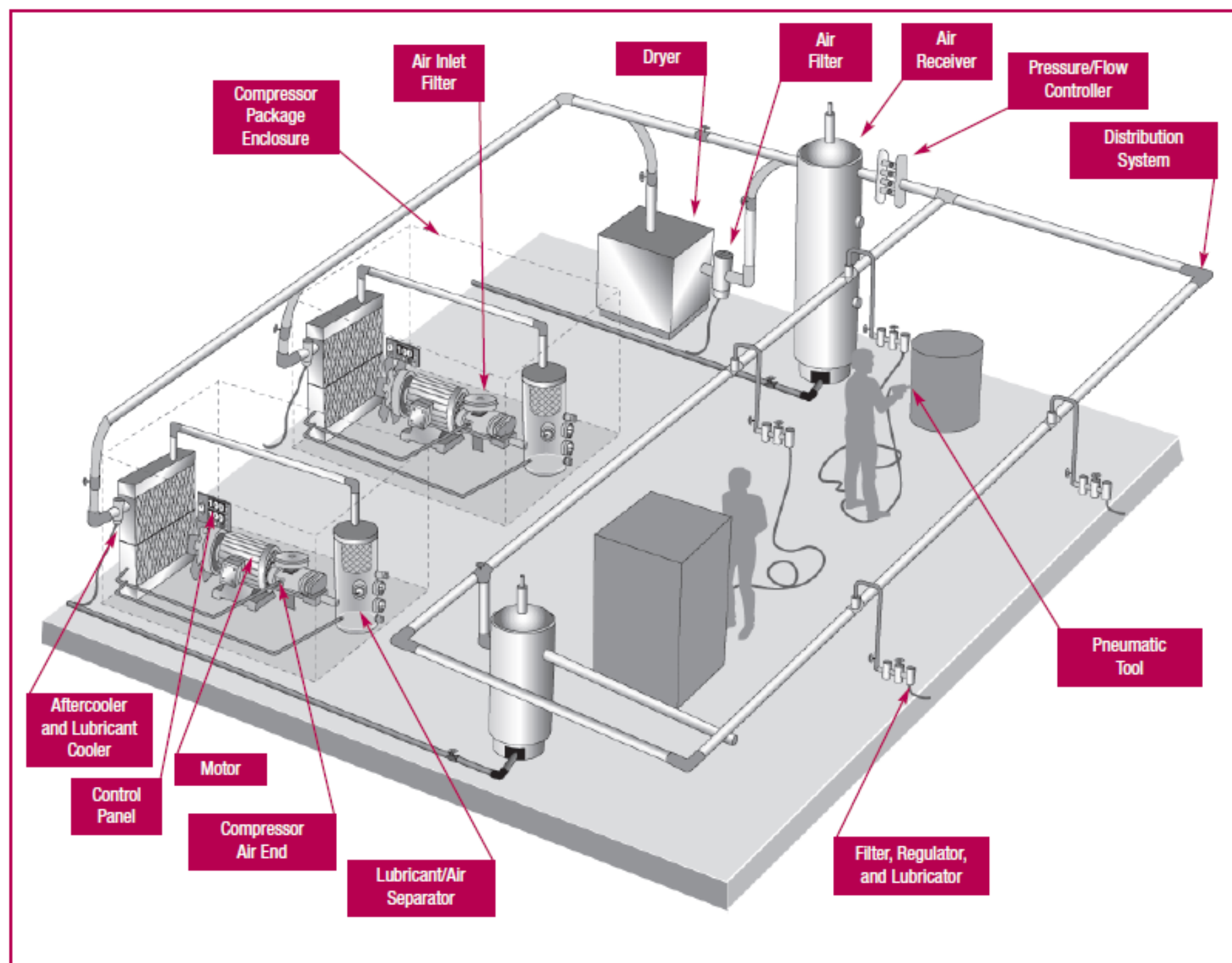
Selection of Compressor

Type of Compressor	Capacity (m ³ /h)		Pressure (bar)	
	From	To	From	To
Roots blower compressor single stage	100	30000	0.1	1
Reciprocating				
- Single / Two stage	100	12000	0.8	12
- Multi stage	100	12000	12.0	700
Screw				
- Single stage	100	2400	0.8	13
- Two stage	100	2200	0.8	24
Centrifugal	600	300000	0.1	450

Comparison of Compressors

Item	Reciprocating	Rotary vane	Rotary Screw	Centrifugal
Efficiency at full load	High	Medium-high	High	High
Efficiency at part load	High due to staging	Poor: below 60% of full load	Poor: below 60% of full load	Poor: below 60% of full load
Efficiency at no load (power as % of full load)	High (10-25%)	Medium (30-40%)	High-poor (25-60%)	High-medium (20-30%)

Compressor air system



Compressor work

$$w_{\text{rev,in}} = \int_1^2 v \, dP$$

Isentropic ($Pv^k = \text{constant}$):

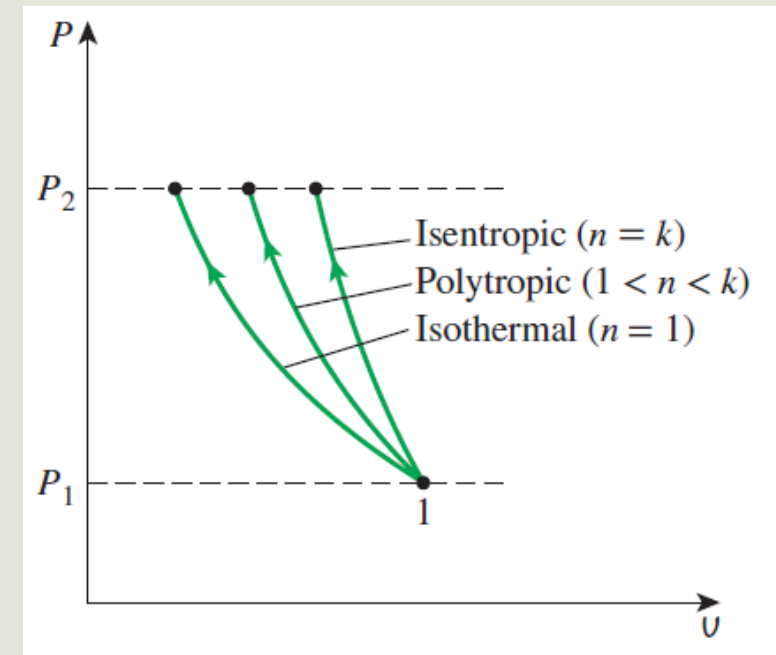
$$w_{\text{comp,in}} = \frac{kR(T_2 - T_1)}{k - 1} = \frac{kRT_1}{k - 1} \left[\left(\frac{P_2}{P_1} \right)^{(k-1)/k} - 1 \right]$$

Polytropic ($Pv^n = \text{constant}$):

$$w_{\text{comp,in}} = \frac{nR(T_2 - T_1)}{n - 1} = \frac{nRT_1}{n - 1} \left[\left(\frac{P_2}{P_1} \right)^{(n-1)/n} - 1 \right]$$

Isothermal ($Pv = \text{constant}$):

$$w_{\text{comp,in}} = RT \ln \frac{P_2}{P_1}$$



The adiabatic compression ($Pv^k = \text{constant}$) requires the maximum work and the isothermal compression ($T = \text{constant}$) requires the minimum.

Multistage Compression with Intercooling

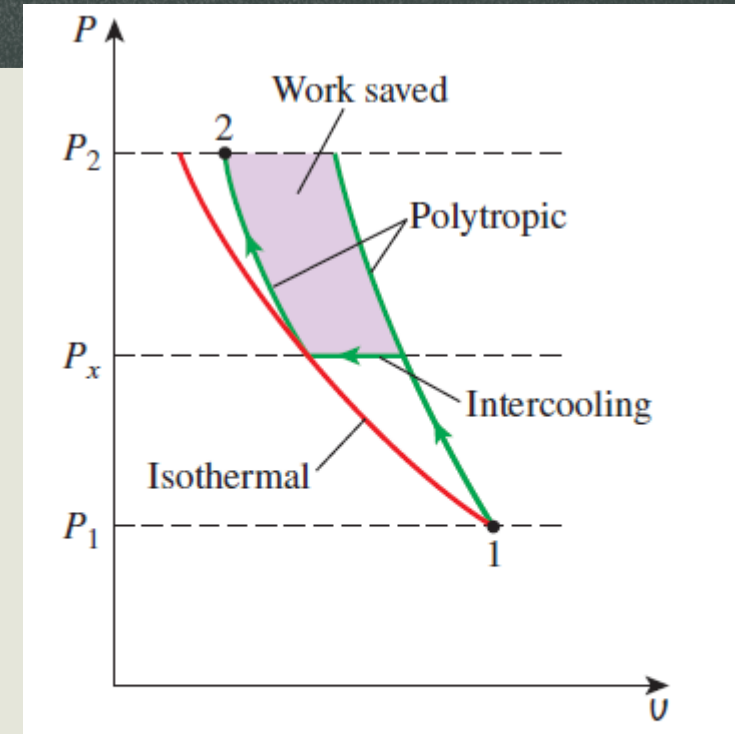
The gas is compressed in stages and cooled between each stage by passing it through a heat exchanger called an *intercooler*.

$$W_{\text{comp,in}} = W_{\text{comp I,in}} + W_{\text{comp II,in}}$$

$$= \frac{nRT_1}{n-1} \left[\left(\frac{P_x}{P_1} \right)^{(n-1)/n} - 1 \right] + \frac{nRT_1}{n-1} \left[\left(\frac{P_2}{P_x} \right)^{(n-1)/n} - 1 \right]$$

$$P_x = (P_1 P_2)^{1/2}$$

To minimize compression work during two-stage compression, the pressure ratio across each stage of the compressor must be the same.



Multistage Compression with Intercooling



Multistage Compression

- Air is compressed steadily by a reversible compressor from an inlet state of 100 kPa and 300 K to an exit pressure of 900 kPa. Determine the compressor work per unit mass for (a) isentropic compression with $k = 1.4$, (b) polytropic compression with $n = 1.3$, (c) isothermal compression, and (d) ideal two-stage compression with intercooling with a polytropic exponent of 1.3.

Isentropic ($Pv^k = \text{constant}$):

$$w_{\text{comp,in}} = \frac{kR(T_2 - T_1)}{k - 1} = \frac{kRT_1}{k - 1} \left[\left(\frac{P_2}{P_1} \right)^{(k-1)/k} - 1 \right]$$

Polytropic ($Pv^n = \text{constant}$):

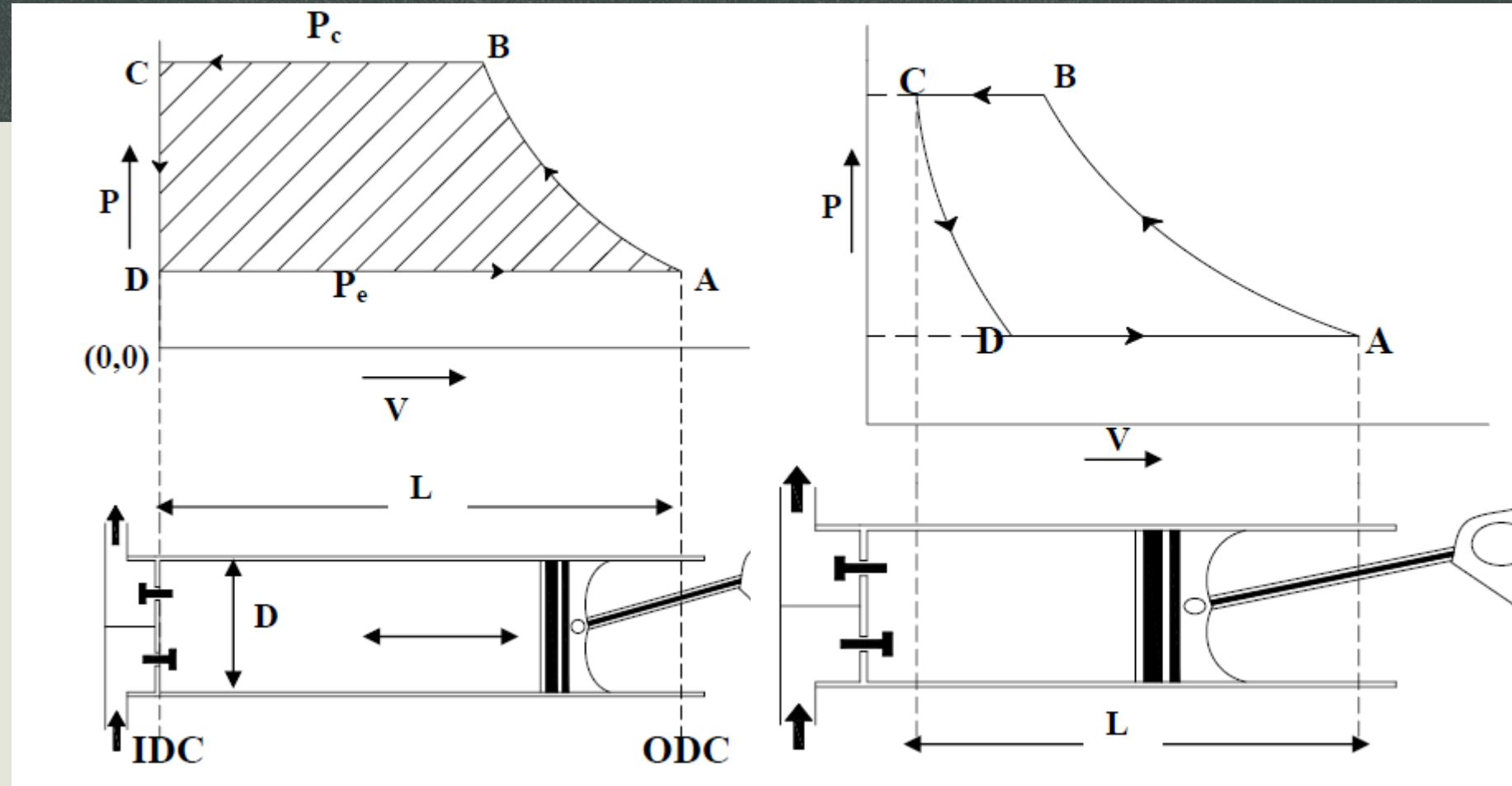
$$w_{\text{comp,in}} = \frac{nR(T_2 - T_1)}{n - 1} = \frac{nRT_1}{n - 1} \left[\left(\frac{P_2}{P_1} \right)^{(n-1)/n} - 1 \right]$$

Isothermal ($Pv = \text{constant}$):

$$w_{\text{comp,in}} = RT \ln \frac{P_2}{P_1}$$

Compressor Performance

$$\eta_{\text{comp}} = \frac{w_{\text{isothermal}}}{w_{\text{actual}}}$$



$$\text{Volumetric Efficiency} = \frac{\text{Free Air Delivered}}{\text{Compressor Displacement}}$$

Efficient Operation of Compressor

- **Cool air intake** – Compressor room always at a high temp – Every 4°C drop – 1% reduction in power – Fresh cool intake with less pressure drop
- **Dust free intake** – need of high eff filter, maintenance - Every 250 mm WC pressure drop due to choked filter increase the power consumption by 2%
- **Dry air intake** – moisture not suitable for pneumatics
- **Location of compressor** – Accessibility of clean, cold and dry air
- **Adequate intercooler** – approach the isothermal compression
- **Adequate after-cooler** – A high temp. and humid air enters the receiver which may lead to condensation – corrosion, pressure drop and leakage in pipe lines
- **Reduced delivery pressure if possible** – lesser the pressure ratio – lesser the power requirement

Efficient Operation of Compressor

- Segregate the low and high pressure requirements – don't use reducing valves if possible
- Minimum pressure drop in the air lines – 0.3 bar in the main line – 0.5 bar in the distribution line
- Avoid misuse of compressed air – like body or floor cleaning – use blower air instead
- Elevation – higher the altitude and higher the power

Altitude Meters	Barometric Pressure milli bar*	Percentage Relative Volumetric Efficiency Compared with Sea Level	
		At 4 bar	At 7 bar
Sea level	1013	100.0	100.0
500	945	98.7	97.7
1000	894	97.0	95.2
1500	840	95.5	92.7
2000	789	93.9	90.0
2500	737	92.1	87.0

Avoid air leakage

- *Manufacturers are quick to identify energy (and thus money) losses from hot surfaces and to insulate those surfaces.*
- Not so sensitive to save compressed air since they view air as being free
- Attention is when the air and pressure losses interfere with the normal operation of the plant.
- Several studies at plants have revealed that up to 40% of the compressed air is lost through leaks.
- Eliminating the air leaks totally is impractical, and a leakage rate of 10% is considered acceptable.
- **Air leak locations:** Joints, flange connections, elbows, reducing bushes, sudden expansions, valve systems, filters, hoses, check valves, relief valves, extensions, and the equipment connected to the compressed-air lines.

Detecting air leaks

- Perhaps the simplest way of detecting a large air leak is **to listen for it**.
- The high velocity of the air escaping the line produces a hissing sound that is difficult not to notice except in environments with a high noise level.
- Another way of detecting air leaks, especially small ones, is **to test the suspected area with soap water and to watch for soap bubbles**.
- This method is obviously not practical for a large system with many connections.
- A modern way of checking for air leaks is **to use an acoustic leak detector**, which consists of a directional microphone, amplifiers, audio filters, and digital indicators.
- **Pressure drop test**

Capacity control

- **ON/OFF control** – using pressure switches – suitable for small compressors
- **Load/Unload** – Compressors may consume power even at unload conditions
- **Multi-step control** – High capacity compressor – 100, 75, 50 and 25%
- **Vane / Speed control** – Centrifugal type

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