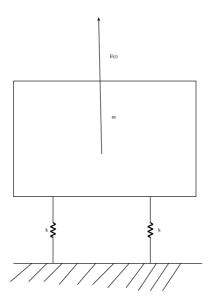
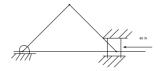
## Gate ME-2018

## AI24BTECH11032 Shreyansh Sonkar

40) A machine of mass m = 200kg is supported on two mounts, each of stiffness  $k = 10\frac{kN}{m}$ . The machine is subjected to an external force (in N)  $F(t) = 50\cos 5t$ . Assuming only vertical translatory motion, the magnitude of the dynamic force (in N) transmitted from each mount to the ground is \_\_\_\_\_\_ (correct to two decimal places).

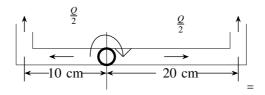


41) A slider crank mechanism is shown in the figure. At some instant, the crank angle is 45° and a force of 40 N is acting towards the left on the slider. The length of the crank is 30 mm and the connecting rod is 70 mm. Ignoring the effect of gravity, friction and inertial forces, the magnitude of the crankshaft torque (in Nm) needed to keep the mechanism in equilibrium is (correct to two decimal places).



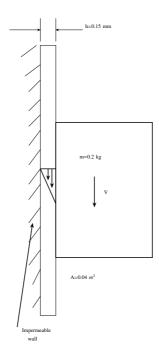
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42) A sprinkler shown in the figure rotates about its hinge point in a horizontal plane due to water flow discharged through its two exit nozzles.



The total flow rate Q through the sprinkler is  $1\frac{litre}{sec}$  and the cross-sectional area of each exit nozzle is 1 cm<sup>2</sup>. Assuming equal flow rate through both arms and a frictionless hinge, the steady state angular speed of rotation (in  $\frac{rad}{s}$ ) of the sprinkler is \_\_\_\_\_\_ (correct to two decimal places).

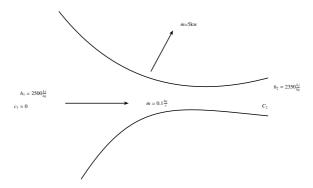
43) A solid block of 2.0 kg mass slides steadily at a velocity V along a vertical wall as shown in the figure below. A thin oil film of thickness h=0.15 mm provides lubrication between the block and the wall. The surface area of the face of the block in contact with the oil film is 0,04 m $^2$ . The velocity distribution within the oil film gap is linear as shown in the figure. Take dynamic viscosity of oil as  $7 \times 10^{-3}$  Pa $_{\rm s}$  and acceleration due to gravity as  $10\frac{m}{s^2}$ . Neglect weight of the oil. The terminal velocity V(in  $\frac{m}{s}$ ) of the block is \_\_\_\_\_ (correct to two decimal places).



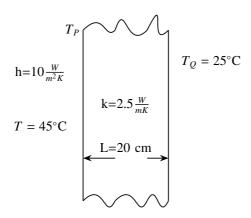
44) A tank of volume contains a mixture of saturated water and saturated steam at  $200^{\circ}$  The mass of the liquid present is 8 kg. The entropy (in  $\frac{kj}{kg}$  k) of the mixture is \_\_\_\_\_ (correct to two decimal places). Property data for saturated steam and water are: At  $200^{\circ}$ C,

$$\begin{split} p_{\text{sat}} &= 1.5538 \, \text{MPa}, \\ v_f &= 0.001157 \, \frac{m^3}{kg}, v_g = 0.12736 \frac{m^3}{kg}, \\ s_{fg} &= 4.1014 \, \frac{kJ}{kg} \, \, \text{K} \, \, , s_f = 2.3309 \frac{kJ}{kg} \, \, \text{K} \, \, . \end{split}$$

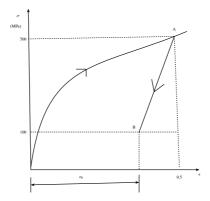
45) Steam flows through a nozzle at a mass flow rate of  $\dot{m} = 0.1 \frac{\text{kg}}{\text{s}}$  with a heat loss of 5kW. The enthalpies at inlet and exit are  $2500 \frac{\text{kJ}}{\text{kg}}$  and  $2350 \frac{\text{kJ}}{\text{kg}}$ , respectively. Assuming negligible velocity at inlet  $(C_1 \approx 0)$ , the velocity  $(C_2)$  of steam (in  $\frac{m}{s}$ ) at the \_\_\_\_\_ (correct to two decimal places).



- 46) An engine working on air standard Otto cycle is supplied with air at 0.1 MPa and 35° The compression ratio is 8. The heat supplied is  $500 \frac{kJ}{kg}$  Property data for air:  $c_p = 1.005 \frac{kJ}{kg}$ ,  $c_v = 0.718 \frac{kJ}{kg}$ ,  $R = 0.287 \frac{kJ}{kg}$  K.The maximum temperature (in K) of the cycle is (correct to two decimal places).
- 47) A plane slab of thickness L and thermal conductivity k is heated with a fluid on one side (P), and the other side (Q) is maintained at a constant temperature,  $T_Q$  of 25°C, as shown in the figure. The fluid is at 45°C and the surface heat transfer coefficient, h, is  $10 \frac{\text{W}}{\text{m}^2\text{K}}$  The steady state temperature,  $T_P$  (in °C) of the side which is exposed to the fluid is \_\_\_\_\_\_ (correct to two decimal places).



48) The true stress  $(\sigma)$  – true strain  $(\epsilon)$  diagram of a strain hardening material is shown in figure. First, there is loading up to point A, up to stress of 500 MPa and strain of 0.5. Then from point A,there is unloading up to point B,i.e., to stress of 100 MPa. Given that the Young's modulus E = 200GPa, the natural strain at point  $B(\epsilon_B)$  is \_\_\_\_\_\_ (correct to two decimal places).



- 49) An orthogonal cutting operation is being carried out in which uncut thickness is 0.010 mm cutting speed is  $130 \frac{\text{m}}{\text{min}}$ , rake angle 15° and width of cut is 6 mm It is observed that the chip thickness is 0.015 mm the cutting force is 60 N and the thrust force is 25 N . The ratio of friction energy to total energy is (correct to two decimal places).
- 50) A bar is compressed to half of its original length. The magnitude of true strain produced in the deformed bar is \_\_\_\_\_\_ (correct to two decimal places).

51) The minimum value of 3x + 5y such that :

$$3x + 5y \le 15$$
$$4x + 9y \le 8$$
$$13x + 2y \le 2$$
$$x \ge 0, y \ge 0$$

is
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52) Processing times (including setup times) and due dates for six jobs waiting to be processed at a work centre are given in the table. The average tardiness (in days) using shortest processing time rule is \_\_\_\_\_\_ (correct to two decimal places).

Job	Processing time (days)	Due date (days)
A	3	8
В	7	16
С	4	4
D	9	18
E	5	17
F	13	19