

Submitted by

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MATLAB version 7.10.0 R2010a

5.3 Solve the problem in example 5.1 using central difference method implemented by a computer program in language of your choice using delta t=0.1sec. Note that this problem was presented in example 5.2 and the results were presented in table 5.2

Solution

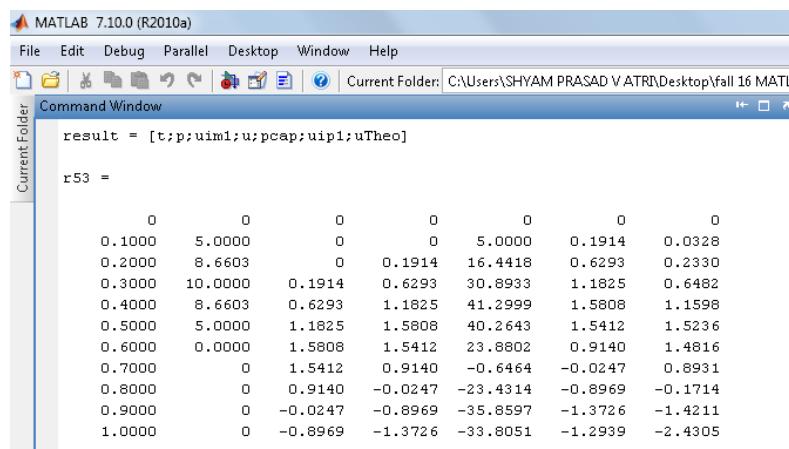
Use the general program written for implementing central difference method

And make the following call

```
% result = [t;p;uim1;u;pcap;uip1;uTheo]= CentralDifference(dt,z,tn)
r53=CentralDifference(0.1,0.05,1)
```

The Theoretical value computed is in close comparison to the numerical values u computed using central difference method.

Here is the result Image



The screenshot shows the MATLAB 7.10.0 (R2010a) interface with the Command Window active. The window title is "Command Window". The current folder is set to "C:\Users\SHYAM PRASAD V ATRI\Desktop\fall 16 MATL". The command entered is "r53 = CentralDifference(0.1,0.05,1)". The output displayed is a matrix "r53" with 11 rows and 7 columns, representing numerical values for time steps from 0.1000 to 1.0000 and spatial points from 0 to 6. The values are as follows:

	0	0	0	0	0	0
0.1000	5.0000	0	0	5.0000	0.1914	0.0328
0.2000	8.6603	0	0.1914	16.4418	0.6293	0.2330
0.3000	10.0000	0.1914	0.6293	30.8933	1.1825	0.6482
0.4000	8.6603	0.6293	1.1825	41.2999	1.5808	1.1598
0.5000	5.0000	1.1825	1.5808	40.2643	1.5412	1.5236
0.6000	0.0000	1.5808	1.5412	23.8802	0.9140	1.4816
0.7000	0	1.5412	0.9140	-0.6464	-0.0247	0.8931
0.8000	0	0.9140	-0.0247	-23.4314	-0.8969	-0.1714
0.9000	0	-0.0247	-0.8969	-35.8597	-1.3726	-1.4211
1.0000	0	-0.8969	-1.3726	-33.8051	-1.2939	-2.4305

5.4 Repeat the problem 5.3 using delta t=0.05sec. How does the time step affect the accuracy of the solution?

The Same function is used with minor small change to time step of 0.05.

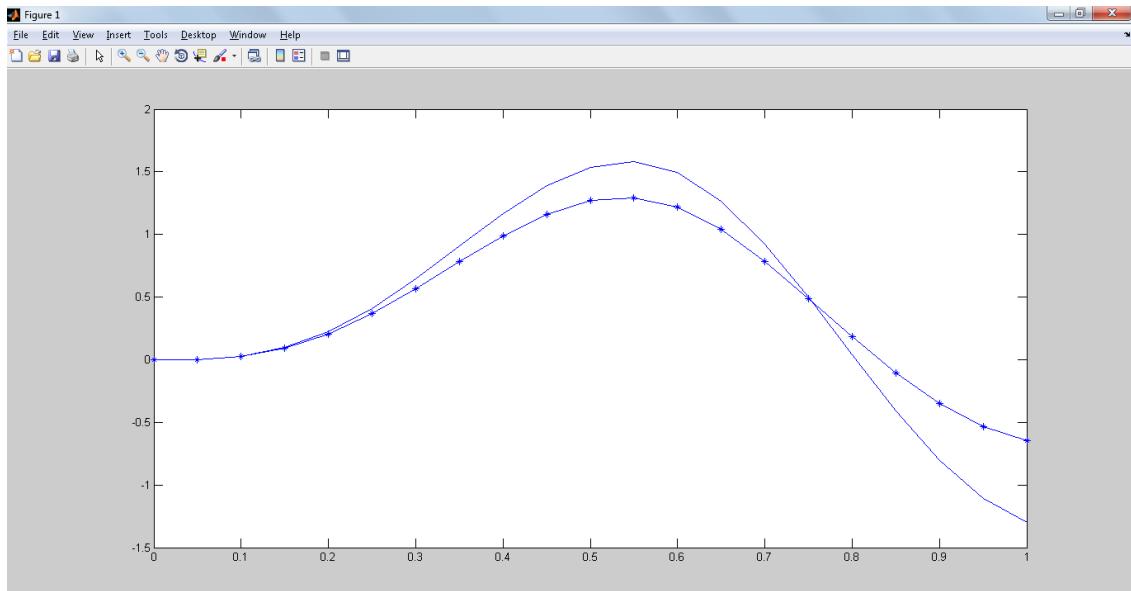
```
clear all
clc
result = [t;p;uim1;u;pcap;uip1;uTheo]= CentralDifference(dt,z,tn)
r54=CentralDifference(0.05,0.05,1)
```

MATLAB 7.10.0 (R2010a)						
File	Edit	Debug	Parallel	Desktop	Window	Help
Current Folder	File	Open	Save	Print	Help	Current Folder: C:\Users\SHYAM PRASAD V ATRI\Desktop\fall 16 MATL
Command Window	File	Edit	Clear	Help	Search	
<pre>result = [t;p;uim1;u;pcap;uip1;uTheo]</pre>						
<pre>r54 =</pre>						
0	0	0	0	0	0	0
0.0500	2.5882	0	0	2.5882	0.0251	0.0042
0.1000	5.0000	0	0.0251	9.8448	0.0957	0.0328
0.1500	7.0711	0.0251	0.0957	22.9915	0.2234	0.1052
0.2000	8.6603	0.0957	0.2234	42.1576	0.4096	0.2330
0.2500	9.6593	0.2234	0.4096	66.2937	0.6442	0.4173
0.3000	10.0000	0.4096	0.6442	93.2415	0.9060	0.6482
0.3500	9.6593	0.6442	0.9060	119.9546	1.1656	0.9053
0.4000	8.6603	0.9060	1.1656	142.8457	1.3880	1.1598
0.4500	7.0711	1.1656	1.3880	158.2194	1.5374	1.3776
0.5000	5.0000	1.3880	1.5374	162.7433	1.5814	1.5236
0.5500	2.5882	1.5374	1.5814	153.9015	1.4955	1.5663
0.6000	0.0000	1.5814	1.4955	130.3785	1.2669	1.4816
0.6500	0	1.4955	1.2669	94.9140	0.9223	1.2568
0.7000	0	1.2669	0.9223	51.3236	0.4987	0.8931
0.7500	0	0.9223	0.4987	4.0943	0.0398	0.4068
0.8000	0	0.4987	0.0398	-42.0720	-0.4088	-0.1714
0.8500	0	0.0398	-0.4088	-82.7222	-0.8038	-0.7981
0.9000	0	-0.4088	-0.8038	-114.0769	-1.1085	-1.4211
0.9500	0	-0.8038	-1.1085	-133.3769	-1.2960	-1.9838
1.0000	0	-1.1085	-1.2960	-139.1195	-1.3518	-2.4305

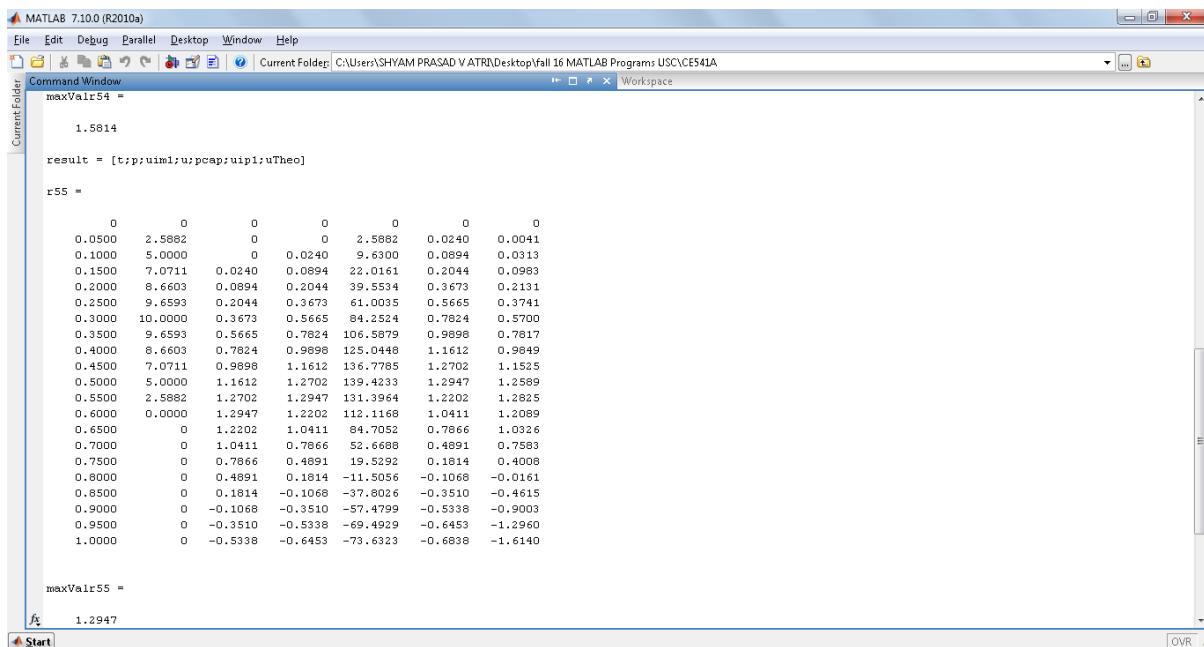
We observe that as the time step is reduced the solution becomes more accurate which is clearly seen from the above table that is as the time step reduces the solution is more close to the theoretical solution.

5.5 An SDF system has the same mass and stiffness as in example 5.1, But the damping ratio is  $\zeta=20\%$ . Determine the response to this system to the excitation as in example 5.1, by the central difference method using delta t=0.05. Plot the response as a function of time, compare the solution with problem 5.3 and comment how the damping affects the peak response.

```
clear all
clc
%result = [t;p;uim1;u;pcap;uip1;uTheo]= CentralDifference(dt,z,tn)
r54=CentralDifference(0.05,0.05,1)
maxValr54=max(r54(:,4))
r55=CentralDifference(0.05,0.2,1)
maxValr55=max(r55(:,4))
plot(r54(:,1),r54(:,4))
hold on
plot(r55(:,1),r55(:,4),'-*')
```



The solution with '\*' depicts the solution for  $z=0.2$



5.6 Solve the problem in example 5.1 by central difference method using delta  $t=1/3$  sec. Carry out the solution to 2 sec, and comment on what happens to the solution and why.

```

clear all
clc
%result = [t;p;uim1;u;pcap;uip1;uTheo]= CentralDifference(dt,z,tn)
r56=CentralDifference(1/3,0.05,2)

```

r56 =						
0	0	0	0	0	0	0
0.3333	9.8481	0	0	9.8481	3.9104	0.8184
0.6667	0	0	3.9104	-21.2749	-8.4477	1.1505
1.0000	0	3.9104	-8.4477	37.9795	15.0806	-2.4305
1.3333	0	-8.4477	15.0806	-64.8062	-25.7328	-0.3851
1.6667	0	15.0806	-25.7328	109.2225	43.3693	3.6182
2.0000	0	-25.7328	43.3693	-183.4351	-72.8371	-1.7509

Here the central difference gives meaning less results as  $\Delta T/T_n=1/3$  which exceeds the stability limit  $1/\pi$ .

The program for all the problems

```
function [result] = CentralDifference(dt,z,tn)
m=0.2533; %kip s2 in-1
k=10; %kips in-1
%tn=1; %s
wn=(k/m)^0.5; %rd s-1
%z=0.05;% zeta
c=2*m*wn*z; % damp coef
%dt=0.1;%s
```

t=0:dt:tn;

p=10\*sin(pi\*t/0.6);

h=1;

```
while(h<=size(t,2) )
    if t(h)>0.6
        p(h)=0;
    end
    h=h+1;
end
```

u=zeros(1,size(t,2));
% uTheo=zeros(1,size(t,2));
pcap=zeros(1,size(t,2));

```
u0=0; % initial displacement
u(1)=u0;
ud0=0; % initial velocity
udd0=(p(1)-c*u0-k*u0)/m;
um1=(u0-dt*ud0+dt^2*0.5*udd0);
```

```
kcap=m/dt^2+c/(2*dt);
a=m/dt^2-c/(2*dt);
b=k-2*m/(dt)^2;
```

pcap(1)=p(1)-a\*um1-b\*u(1);

for i=2:size(t,2)

u(i)=pcap(i-1)/kcap;

```

pcap(i)=p(i)-a*u(i-1)-b*u(i);

end

uim1=[um1 u(1:(size(t,2)-1))];
up1=pcap(size(t,2))/kcap;
uip1=[u(2:(size(t,2))) up1];

% Theoretical values computed using first principles
% w=pi/0.6;
% omg=w/wn;
% C=(10/k)*((1-omg^2)/((1-omg^2)^2+(2*z*omg)^2));
% D=(10/k)*((-2*z*omg)/((1-omg^2)^2+(2*z*omg)^2));
% for k=1:size(t,2)
%
%     uTheo(1,k)=C*sin(w*t(k))+D*cos(w*t(k));
% end

% The theoretical Value is calculated
% usin Duhamels integral
syms fA wfA tA wA zA tauA k
%vA=int((fA*sin(wfA*tauA)*(wA/(k*(1-zA^2)^0.5))*(exp(-zA*wA*(tA-tauA)))*(sin(wA*(tA-tauA)*(1-zA^0.5))),tauA,0,tA);
%vA=int((fA*sin(wfA*tauA)*(wA/(k*(1-zA^2)^0.5))*(exp(-zA*wA*(tA-tauA)))*(sin(wA*(tA-tauA)*(1-zA^0.5))),tauA,0,tA);

% differential equation is
% mxdd+cxd+kx= 10 sin(pi*t/0.6)
vA=int((fA*sin(wfA*tauA/0.6)*(wA/(k*(1-zA^2)^0.5))*(exp(-zA*wA*(tA-tauA)))*(sin(wA*(tA-tauA)*(1-zA^2)^0.5))),tauA,0,tA);

k=10;
fA=10;
wA=2*3.14;
-zA=z;
wfA=pi;
tA=0:dt:tn;
res=eval(vA);

t=0:dt:tn;
disp('result = [t;p;uim1;u;pcap;uip1;uTheo]')

result=[t;p;uim1;u;pcap;uip1;res];
%plot(t,u,t,res)

end
%% %% % calling the function
clear all
clc
%result = [t;p;uim1;u;pcap;uip1;uTheo]= CentralDifference(dt,z,tn)
r53=CentralDifference(0.1,0.05,1)
r54=CentralDifference(0.05,0.05,1)
maxValr54=max(r54(:,4))
r55=CentralDifference(0.05,0.2,1)

```

```

maxValr55=max(r55(:,4))
r56=CentralDifference(1/3,0.05,2)
plot(r54(:,1),r54(:,4))
hold on
plot(r55(:,1),r55(:,4),'-*')

```

6.11 a. A full water tank is supported on a 80ft cantilever tower. It is idealized as an SDF system with weight  $w=100$ kips, lateral stiffness  $k=4$ kips/in, the damping ratio is  $\zeta=5\%$ . The tower supporting the tank is to be designed for ground motion characterised by the design spectrum of fig 6.9.5 scaled to 0.5g peak ground acceleration. Determine the design values of lateral deformation and base shear.

b. The deformation computed for the system in part a seemed excessive to the structural designer, who decided to stiffen the structure by increasing the size. Determine the design values of deformation and base shear for the modified system if its lateral stiffness is 8kips/in. Assume the damping ratio is still 5%. Comment on how stiffening of the system has affected the design requirements. What is the disadvantage of stiffening of the system?

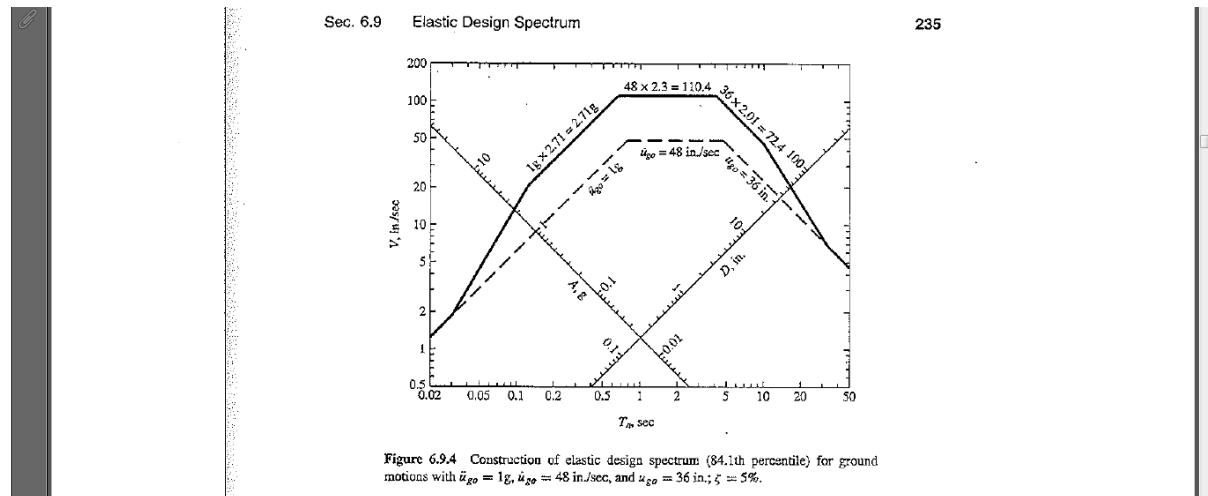


Figure 6.9.4 Construction of elastic design spectrum (84.1th percentile) for ground motions with  $\ddot{u}_{go} = 1$ g,  $\dot{u}_{go} = 48$  in/sec, and  $u_{go} = 36$  in;  $\zeta = 5\%$ .

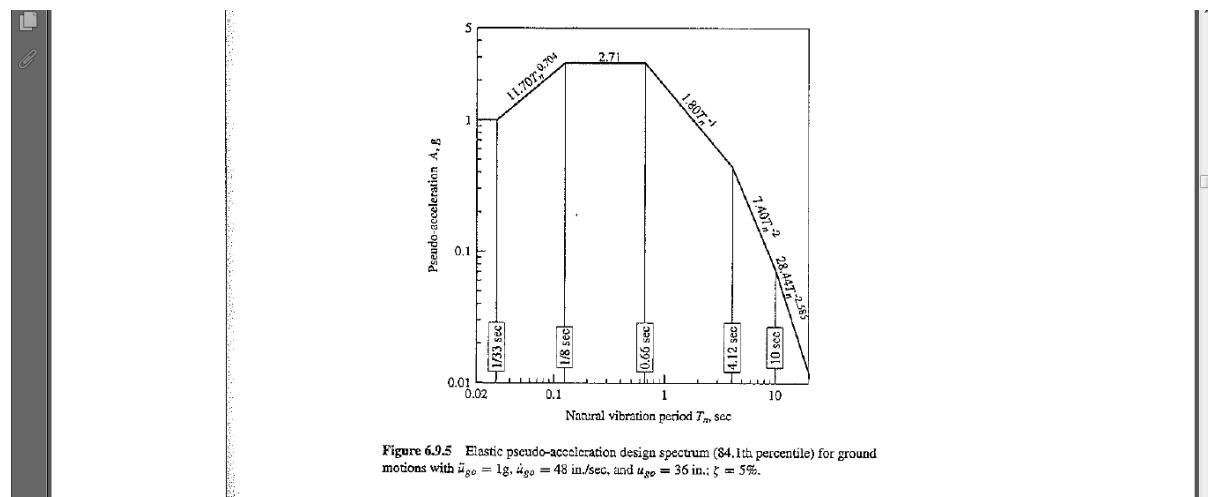
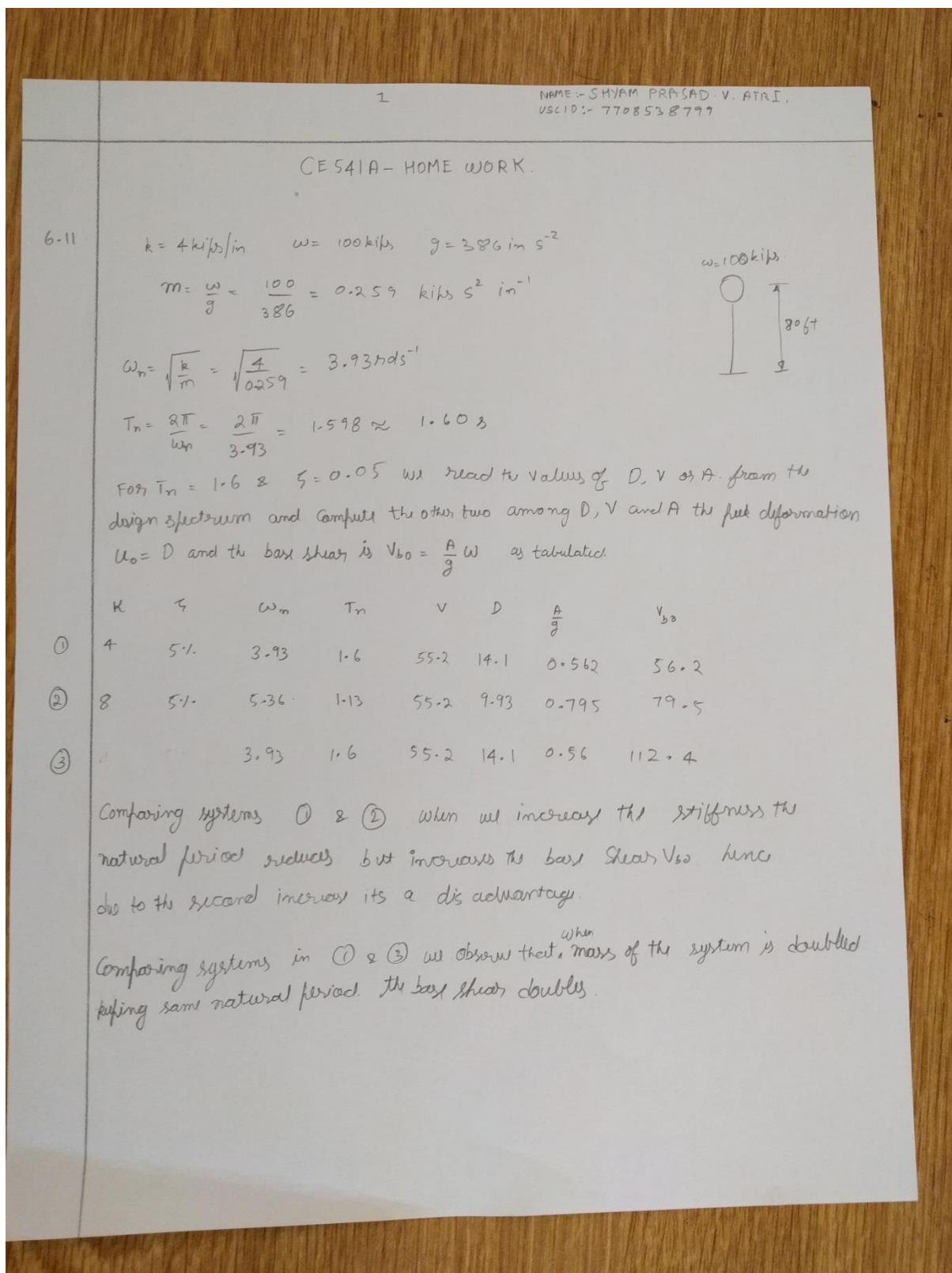


Figure 6.9.5 Elastic pseudo-acceleration design spectrum (84.1th percentile) for ground motions with  $\ddot{u}_{go} = 1$ g,  $\dot{u}_{go} = 48$  in/sec, and  $u_{go} = 36$  in.;  $\zeta = 5\%$ .

Using these diagrams to get values of V,D,A/g



6.18 A one story steel frame of 24ft span and 12 ft height has following properties: The second moment of cross sectional area for beam and columns are  $I_b=160 \text{ in}^4$  and  $I_c=320 \text{ in}^4$  respectively;

the elastic modulus for steel is  $30E3$  ksi. For purposes of dynamic analysis the frame is considered as massless with a weight of 100kips lumped at the beam level; the columns are clamped at the base; the damping ratio is estimated at 5%. Determine the peak values of lateral displacement at the beam level and bending moment throughout the frame due to the design spectrum of fig 6.9.5 scaled to the peak ground acceleration of  $0.5g$ .

Using this chart to get A

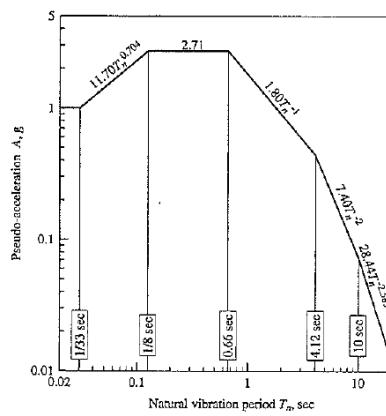
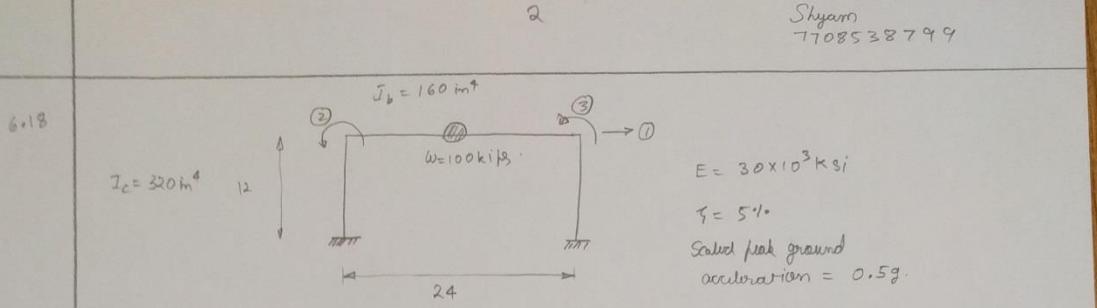


Figure 6.9.5 Elastic pseudo-acceleration design spectrum (84.1th percentile) for ground motions with  $\bar{u}_{go} = 1g$ ,  $\dot{u}_{go} = 48 \text{ in/sec}$ , and  $u_{go} = 36 \text{ in}$ ;  $\zeta = 5\%$ .

Check the images below.!



Using Static Condensation method we determine the lateral stiffness of the structure.  
as in chapter 1

$$k = \frac{EI_c}{h^3} \begin{bmatrix} 24 & 6h & 6h \\ 6h & 5h^2 & h^2/2 \\ 6h & h^2/2 & 5h^2 \end{bmatrix} \begin{bmatrix} k_{tt} & k_{to} \\ k_{to}^T & k_{oo} \end{bmatrix}$$

$$\frac{EI_c}{h^3} \begin{bmatrix} 24 & 6h & 6h \\ 6h & 5h^2 & h^2/2 \\ 6h & h^2/2 & 5h^2 \end{bmatrix} \begin{Bmatrix} u_1 \\ u_2 \\ u_3 \end{Bmatrix} = \begin{Bmatrix} f_s \\ 0 \\ 0 \end{Bmatrix}$$

applying static condensation.

$$k = k_{tt} - k_{to} k_{oo}^{-1} k_{to}^T$$

$$k = 24 \frac{EI_c}{h^3} - \frac{EI_c}{h^3} [6h \ 6h] \begin{bmatrix} 5h^2 & h^2/2 \\ h^2/2 & 5h^2 \end{bmatrix}^{-1} \begin{bmatrix} 6h \\ 6h \end{bmatrix}$$

$$k = \frac{120}{11} \frac{EI}{h^3} = \frac{120}{11} \times \frac{(30 \times 10^3)(320)}{(12 \times 12)^3} = 35.07 \text{ kN/m}^{-1}$$

$$m = \frac{w}{g} = \frac{100}{386} = 0.2591 \text{ kN s}^2 \text{ m}^{-1}$$

$$\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{35.07}{0.2591}} = 11.64 \text{ rad/s}^{-1}$$

$$T_n = \frac{2\pi}{\omega_n} = \frac{2\pi}{11.64} = 0.5408$$

Using the design spectrum chart

$$A = 0.5 \times 2.71 \text{ g} = 1.355 \text{ g}$$

$$D = \frac{A}{\omega_n^2} = \frac{1.355}{\omega_n^2} \text{ g} = \frac{1.355 \times 386}{11.64^2} = 3.86 \text{ in}$$

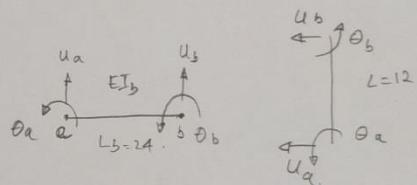
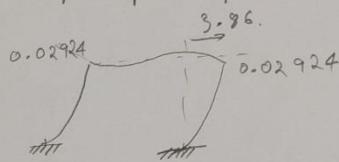
The peak displacement is

$$U_{10} = 3.86 \text{ in}$$

To determine the bending moments we require the joint rotations

$$\begin{Bmatrix} u_{20} \\ u_{30} \end{Bmatrix} = \frac{12}{-11h} \begin{Bmatrix} 1 \\ 1 \end{Bmatrix} u_0 = -\frac{12}{11 \times 12 \times 12} \begin{Bmatrix} 1 \\ 1 \end{Bmatrix} \times 386 \\ = -0.02924 \begin{Bmatrix} 1 \\ 1 \end{Bmatrix}$$

The deformed shape at peak displacement is



For Column

$$M_a = \frac{4EI_c}{L_c} \theta_a + \frac{2EI_c}{L_c} \theta_b + \frac{6EI_c}{L_c^2} u_a - \frac{6EI_c}{L_c^2} u_b$$

$$M_b = \frac{2EI_c}{L_c} \theta_a + \frac{4EI_c}{L_c} \theta_b + \frac{6EI_c}{L_c^2} u_a - \frac{6EI_c}{L_c^2} u_b.$$

using  $\theta_a = 0$ ,  $\theta_b = -0.02924$ ,  $u_a = 0$  and  $u_b = -3.86$

and substituting values for  $E, I_c$  and  $L_c$  gives.

$$M_a = 6824 \text{ kinh-in} \quad M_b = 2925 \text{ kinh-in.}$$

For beam

$$M_a = \frac{4EI_b}{L_b} \theta_a + \frac{2EI_b}{L_b} \theta_b + \frac{6EI_b}{L_b^2} u_a - \frac{6EI_b}{L_b^2} u_b$$

$$M_b = \frac{2EI_b}{L_b} \theta_a + \frac{4EI_b}{L_b} \theta_b + \frac{6EI_b}{L_b} u_a - \frac{6EI_b}{L_b^2} u_b$$

using  $\theta_a = \theta_b = -0.02924$   $u_a = u_b = 0$  and using  $E I_b$  &  $L_b$

gives  $M_a = M_b = -2925$  kih in.

