

1. Estimate the supporting force system at the end A for the cantilever beam shown in Fig. 1. Neglect the weight of the beam.

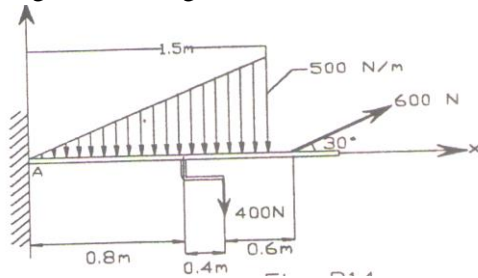


Figure 1

Solution:

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$$\sum F_x = 0$$

$$\Rightarrow A_x + 600 \cos 30^\circ = 0$$

$$\Rightarrow A_x = -519.615 \text{ N}$$

$$\sum F_y = 0$$

$$\Rightarrow A_y - 375 - 400 + 600 \sin 30^\circ = 0$$

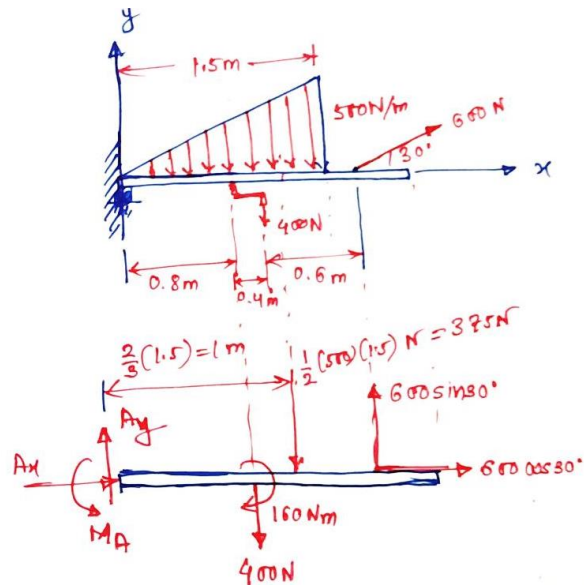
$$\Rightarrow A_y = 775 - 300 = 475 \text{ N}$$

$$\sum M_A = 0$$

$$\Rightarrow M_A - 160 - (0.8)(400) - 1(375) + 1.8(600 \sin 30^\circ) = 0$$

$$\Rightarrow M_A = 160 + 320 + 375 - 540$$

$$= 315 \text{ N}\cdot\text{m}$$



2. Locate the centroid of the shaded area shown in Fig. 2.

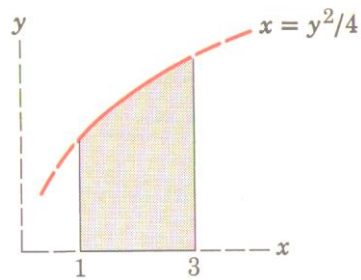


Figure 2

$$27) dA = y dx = 2\sqrt{x} dx$$

$$A = \int_1^3 2\sqrt{x} dx = 2 \cdot \frac{2}{3} x^{3/2} \Big|_1^3$$

$$= \frac{4}{3} (3^{3/2} - 1)$$

$$= 5.595$$

$$\bar{x}_c = \frac{\int \bar{x} dA}{\int dA} = \frac{\int_1^3 2x^{3/2} dx}{5.595}$$

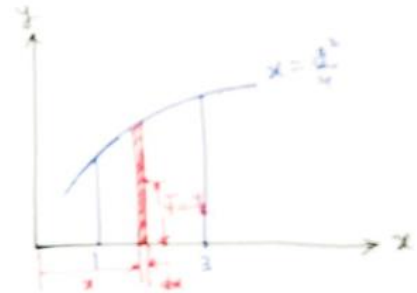
$$= \frac{2}{5.595} \cdot \frac{2}{5} (3^{5/2} - 1)$$

$$\bar{x}_c = \underline{\underline{2.086 \text{ m}}}$$

$$\bar{y}_c = \frac{\int \bar{y} dA}{\int dA} = \frac{\int_1^3 \frac{y}{2} dA}{5.595} = \frac{\frac{1}{2} \int_1^3 y^2 dx}{5.595} = \frac{2 \int_1^3 x dx}{5.595}$$

$$= \frac{2}{5.595} \cdot \frac{1}{2} (3^2 - 1) = \frac{8}{5.595}$$

$$\bar{y}_c = \underline{\underline{1.43 \text{ m}}}$$



3. The stock mounted on the lathe is subjected to a force of 60N. Determine the coordinate direction angle and express the force as a Cartesian vector.

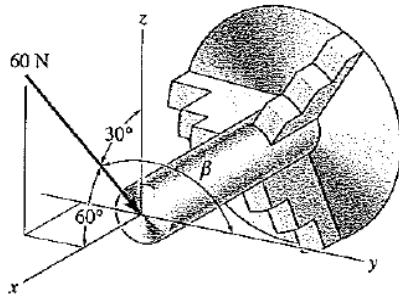


Figure 3

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$$\begin{aligned}\vec{F} &= 60 [\cos \alpha \hat{i} + \cos \beta \hat{j} + \cos \gamma \hat{k}] \\ &= 60 [\cos 60 \hat{i} + F_y \hat{j} + \cos 30 \hat{k}] \\ \vec{F} &= 30 \hat{i} + F_y \hat{j} + 52 \hat{k}\end{aligned}$$

Given that, $|\vec{F}| = 60$

$$\Rightarrow 3600 = 900 + F_y^2 + 52^2$$

$$\Rightarrow F_y = 0$$

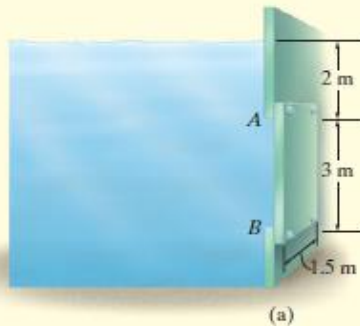
$$\cos \beta = \frac{F_y}{F} = 0$$

$$\Rightarrow \beta = \underline{\underline{90^\circ}}$$

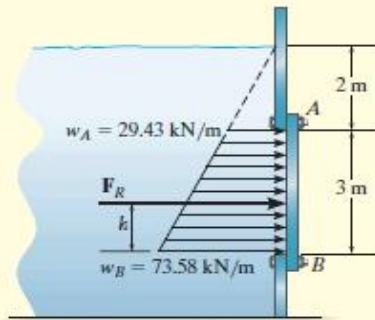
$$\boxed{\vec{F} = 30 \hat{i} + 52 \hat{k}}$$

4. Determine the magnitude and location of the resultant hydrostatic force acting on the submerged rectangular plate AB shown in Fig. 4. The plate has a width of 1.5 m density = 1000 kg/m^3 Fig 4?

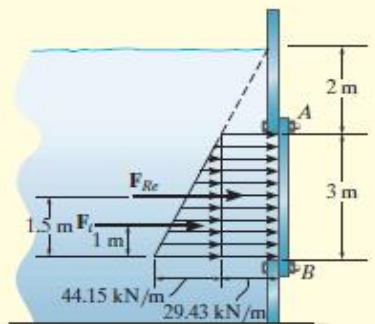
EXAMPLE 9.14



(a)



(b)



(c)

Fig. 9-28

Determine the magnitude and location of the resultant hydrostatic force acting on the submerged rectangular plate AB shown in Fig. 9-28a. The plate has a width of 1.5 m; $\rho_w = 1000 \text{ kg/m}^3$.

SOLUTION I

The water pressures at depths A and B are

$$p_A = \rho_w g z_A = (1000 \text{ kg/m}^3)(9.81 \text{ m/s}^2)(2 \text{ m}) = 19.62 \text{ kPa}$$

$$p_B = \rho_w g z_B = (1000 \text{ kg/m}^3)(9.81 \text{ m/s}^2)(5 \text{ m}) = 49.05 \text{ kPa}$$

Since the plate has a constant width, the pressure loading can be viewed in two dimensions as shown in Fig. 9-28b. The intensities of the load at A and B are

$$w_A = b p_A = (1.5 \text{ m})(19.62 \text{ kPa}) = 29.43 \text{ kN/m}$$

$$w_B = b p_B = (1.5 \text{ m})(49.05 \text{ kPa}) = 73.58 \text{ kN/m}$$

From the table on the inside back cover, the magnitude of the resultant force F_R created by this distributed load is

$$F_R = \text{area of a trapezoid} = \frac{1}{2}(3)(29.4 + 73.6) = 154.5 \text{ kN} \quad \text{Ans.}$$

This force acts through the centroid of this area,

$$h = \frac{1}{3} \left(\frac{2(29.43) + 73.58}{29.43 + 73.58} \right) (3) = 1.29 \text{ m} \quad \text{Ans.}$$

measured upward from B , Fig. 9-31b.

SOLUTION II

The same results can be obtained by considering two components of F_R , defined by the triangle and rectangle shown in Fig. 9-28c. Each force acts through its associated centroid and has a magnitude of

$$F_{Re} = (29.43 \text{ kN/m})(3 \text{ m}) = 88.3 \text{ kN}$$

$$F_t = \frac{1}{2}(44.15 \text{ kN/m})(3 \text{ m}) = 66.2 \text{ kN}$$

Hence,

$$F_R = F_{Re} + F_t = 88.3 + 66.2 = 154.5 \text{ kN} \quad \text{Ans.}$$

The location of F_R is determined by summing moments about B , Fig. 9-28b and c, i.e.,

$$\zeta + (M_R)_B = \Sigma M_B; (154.5)h = 88.3(1.5) + 66.2(1)$$

$$h = 1.29 \text{ m} \quad \text{Ans.}$$

NOTE: Using Eq. 9-14, the resultant force can be calculated as $F_R = \gamma \bar{z} A = (9810 \text{ N/m}^3)(3.5 \text{ m})(3 \text{ m})(1.5 \text{ m}) = 154.5 \text{ kN}$.