

# **TRAXXAS SLASH 4X4 VXL**

**ME49601 TOOLS/METHODS FOR ITERATIVE PRODUCT DESIGN & ANALYSES**

**Team Trophy Truck**

**Jonathan Jackson, Connor Day, Sidh Gurnani, William Snyder**

**School of Mechanical Engineering, Purdue University**



# Outline

- Team Organization and Collaboration
  - File Structures
  - Team organization
  - Roles and Responsibilities
- CAD Overview
  - CAD structure
  - Wireframe and Shared Parameters
  - Surface and Solid Modeling
  - Mass Properties
- Analysis
  - Component and Assembly FEA
  - Topology Optimization
  - CFD
- Visualization

# *TEAM ORGANIZATION AND COMMUNICATION*



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# *Team Organization and Communication*

## File Structures

- OneDrive Shared folder with all files
- Files transfer to other softwares:
  - FEA → STEP files
  - CFD → Parasolid Files

Name	Status	Date modified	Type	S
📁 CAD	🕒 R	12/4/2024 10:36 AM	File folder	
📁 CFD	🕒 R	12/4/2024 10:36 AM	File folder	
📁 FEA	🕒 R	12/3/2024 4:29 PM	File folder	
📁 Part Measurements	🕒 R	12/4/2024 10:36 AM	File folder	
📁 Project Documents	🕒 R	12/4/2024 10:36 AM	File folder	
📁 Visualization	🕒 R	12/4/2024 10:36 AM	File folder	
WORD CAD Structure Guide	🕒 R	11/14/2024 6:37 PM	Microsoft Word D...	
EXCEL To-Do List and Schedule	🕒 R	12/4/2024 10:21 AM	Microsoft Excel W...	

# Team Organization and Communication

## Team Organization

### ▪ Item Checklist

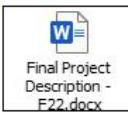
- Helped with accountability and progress tracking
- Schedule to keep all deadlines and relevant information in one place
- Text Group chat for easy communication

### ▪ Parts List and BOM:

- Helps keep track of parts and also ensures the team doesn't overlook parts during CAD and Analysis
- Not all hardware is included in BOM, but instructions on where to keep hardware are provided

### ▪ Team Standards:

- Standardization of units, naming conventions
- Text Group chat with reminders

Assignment	Due Date	Notes
RC Car Progress Presentation 1	11/11/2024	 RC Car Vehicle Project Design Review #1.docx
Full Parametric Car CAD	11/16/2024	Linked CAD
Car Parts List	11/1/2024	Track all parts. VIDEO teardown
Car Component Weight	11/1/2024	weights in BOM
RC Car Progress Presentation 2	11/18/2024	Will link doc when posted
RC Car Progress Presentation 3	11/25/2024	Will link doc when posted
Semester Project Presentation	12/4/2024	 Final Project Description - F22.docx
Turn In USB drive of shared files	12/4/2024	

To Do Item	Due Date	Assigned to	Started	Completed	Checked Off
Create To Do list	10/28/2024	Connor	Y	Y	SG
Update Schedule	10/28/2024	Connor	Y	Y	SG
Initial Shell CAD	11/6/2024	Sidh	Y	Y	CD
Shell CFD	11/9/2024	Sidh	Y	Y	CD
Initial Chassis CAD	11/6/2024	Jackson	Y	Y	CD
Chassis Torsion FEA	11/9/2024	Jackson	Y	Y	CD
Drivetrain CAD	11/6/2024	Connor	Y	Y	CD
MOI/Torque Analysis for bigger tires	11/9/2024	Connor	N	N	
Initial Suspension CAD	11/6/2024	William	Y	Y	CD
Suspension Motion Study	11/9/2024	William	N	N	
Weigh Components and Parts List	11/1/2024	Connor	Y	Y	WS
Prelim Render/Visualization	11/10/2024	Sidh	Y	Y	CD
Main Assm Skeleton Sketch	11/12/2024	Sidh	Y	Y	CD
Driveshaft Torque FEA	11/18/2024	Connor	Y	Y	SG
Differential Cases Modeling	11/21/2024	Connor	Y	Y	JJ
Final Chassis CAD	11/21/2024	Jackson	Y	Y	CD
Final Shell CAD	11/21/2024	Sidh	Y	Y	CD
Gear Torque Topology Optimization	11/17/2024	Connor	Y	Y	CD
Rear Suspension modeling	12/4/2024	William	Y	Y	CD
Full assembly visualization	12/4/2024	All	Y	Y	CD
Optimization screenshots	12/4/2024	Connor	Y	Y	SG
FEA/CFD files and screenshots	12/4/2024	Connor	Y	Y	SG
All screenshots in presentation	12/4/2024	All	Y	Y	CD
Finish presentation	12/4/2024	All	Y	Y	CD
Final Chassis Top Optimization	12/4/2024	Connor	Y	Y	JJ
Final Chassis FEA	12/4/2024	Connor	Y	Y	JJ
FEA of bottom support and Top rod	12/4/2024	William	Y	Y	CD
Tire Tread	12/4/2024	William	Y	Y	CD
Custom Livery	12/4/2024	Sidh	Y	Y	CD
Front Bumper FEA	12/4/2024	Connor	Y	Y	WS
Rear Bumper FEA	12/4/2024	Connor	Y	Y	WS
Control Arm Topology Optimization	12/4/2024	Connor	Y	Y	WS
Presentation and Flash Drive Submission	12/4/2024	All	Y	N	

# *Team Organization and Communication*

## Roles and Responsibilities

Name	Responsibility
Jonathan Jackson	Chassis
Connor Day	Drivetrain/FEA & Topology
Sidh Gurnani	Primary Shell Surface/CFD
William Snyder	Front/Rear Suspension

# CAD OVERVIEW



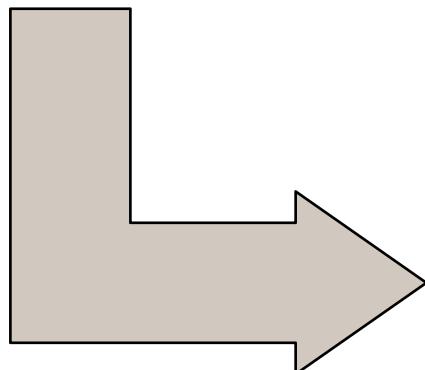
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# CAD Overview

## CAD Structure

Top-Down  
Skeleton



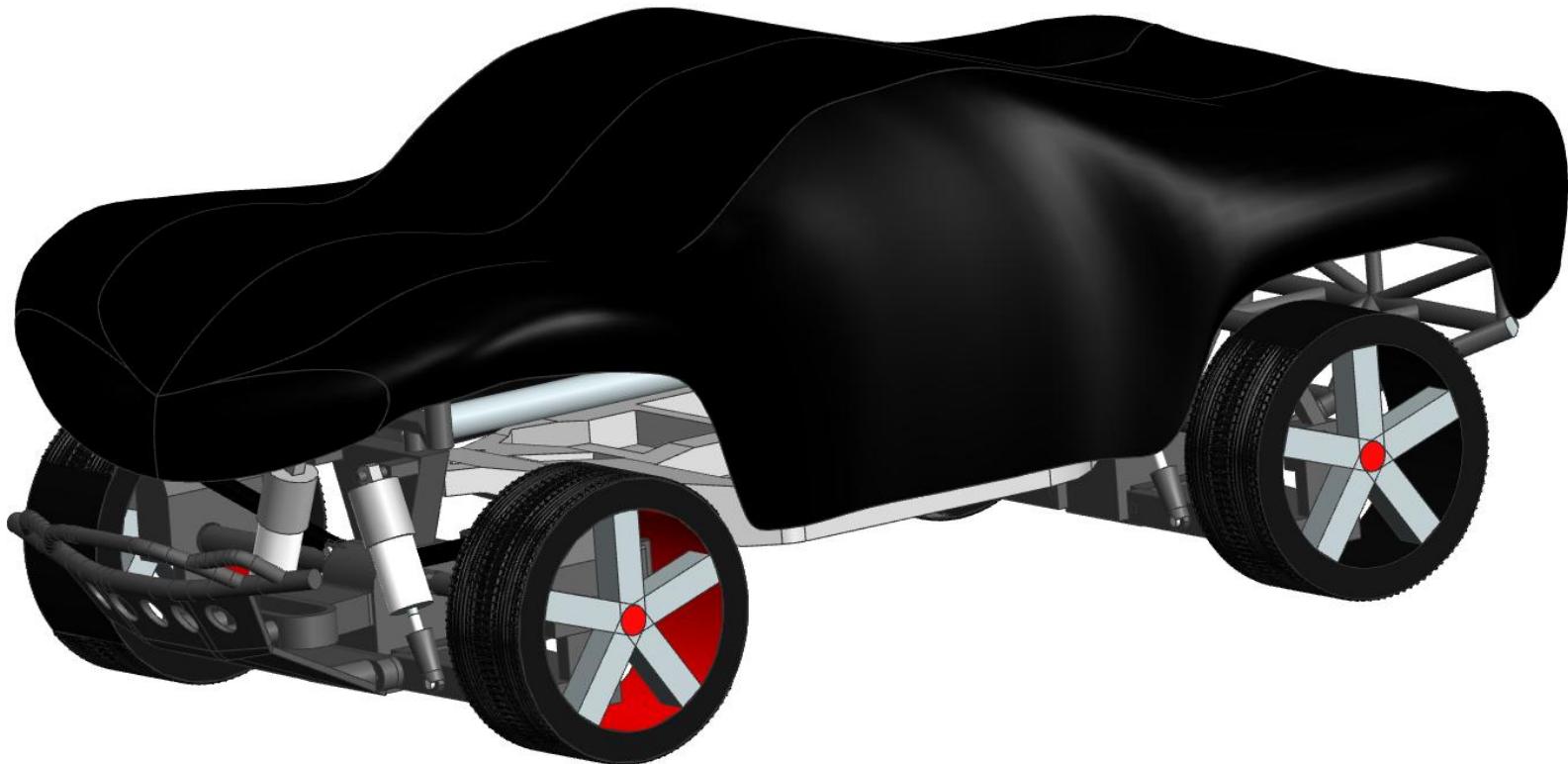
Subsystems

- Overall System Geometry
- Expressions

- Shell
- Drivetrain
- Chassis
- Suspension



# *Overall CAD*



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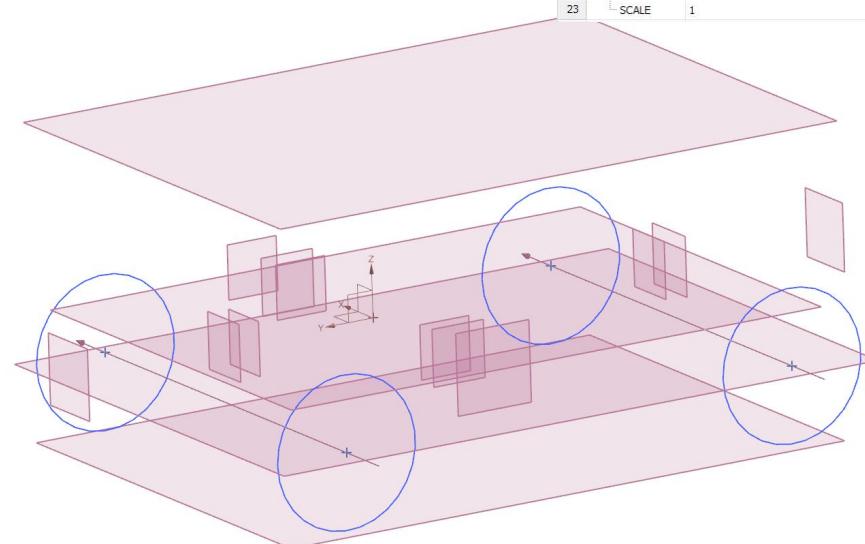
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# CAD Overview

## Wireframe and Shared Parameters

- Global Scale Factor
- Accurate Datum Planes/Wireframe model for sub-assembly references
  - Relevant geometry from skeleton and wireframes wavelinked into parts and assemblies as needed
  - Linked parameters were also used to share dimensions

	Name	Formula	Value	Units	Dimensionality	Type
1	Default Group					
2				in	Length	Number
3	p2	3.80984114*SCALE	3.80984114	in	Length	Number
4	p5	12*SCALE	12	in	Length	Number
5	p8	3.5*SCALE	3.5	in	Length	Number
6	p11	3.5*SCALE	3.5	in	Length	Number
7	p14	0.65*SCALE	0.65	in	Length	Number
8	p17	0.65*SCALE	0.65	in	Length	Number
9	p20	11.7/2*SCALE	5.85	in	Length	Number
10	p23	11.7/2*SCALE	5.85	in	Length	Number
11	p26	0.6*SCALE	0.6	in	Length	Number
12	p29	0.6*SCALE	0.6	in	Length	Number
13	p32	0	0	in	Length	Number
14	p35	3.5*SCALE	3.5	in	Length	Number
15	p38	4.08/2*SCALE	2.04	in	Length	Number
16	p39	4.08*SCALE	4.08	in	Length	Number
17	p40	4.08*SCALE	4.08	in	Length	Number
18	p94	4.08*SCALE	4.08	in	Length	Number
19	p96	4.08*SCALE	4.08	in	Length	Number
20	p99	(22.4-13.2)/2*SCALE	4.6	in	Length	Number
21	p102	22.4*SCALE	22.4	in	Length	Number
22	p105	4.9*SCALE	4.9	in	Length	Number
23	SCALE	1	1		Unitless	Number



# CAD Overview

## Primary Surface Modeling



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# CAD Overview

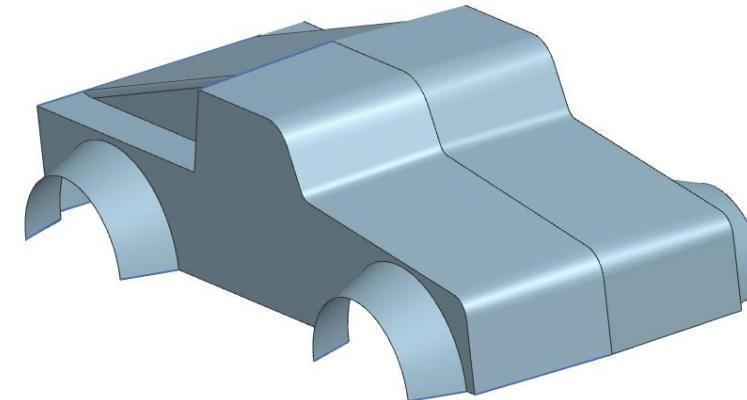
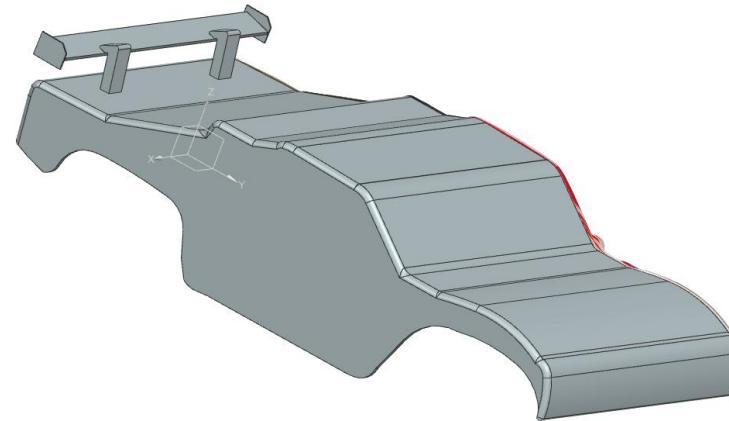
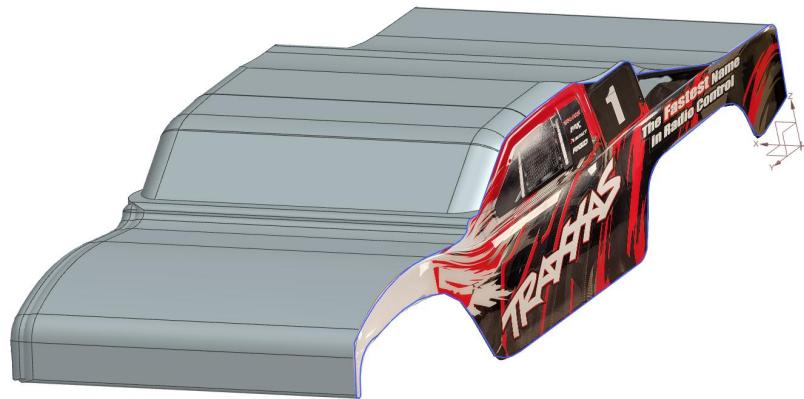
## Primary Surface Modeling

- Slightly modified from Traxxas's
  - Overall tighter packaging and reduction of sharp corners throughout surface
  - Creates smoother surface overall



# CAD Overview

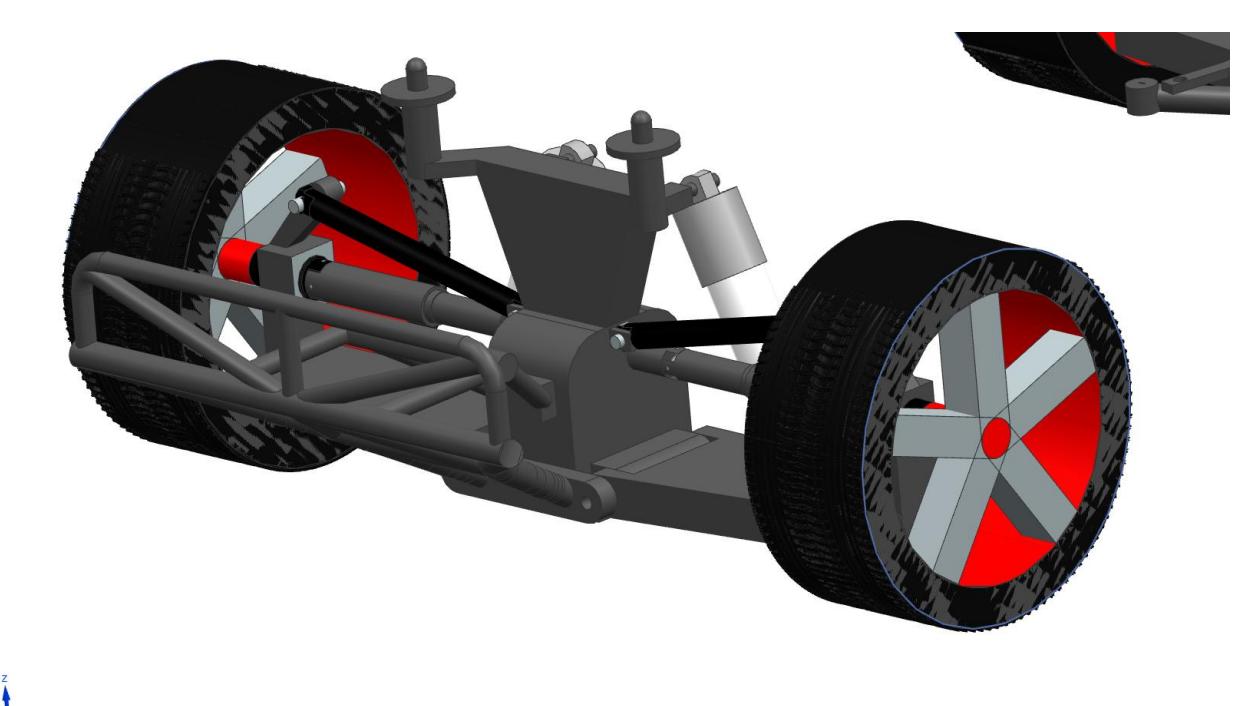
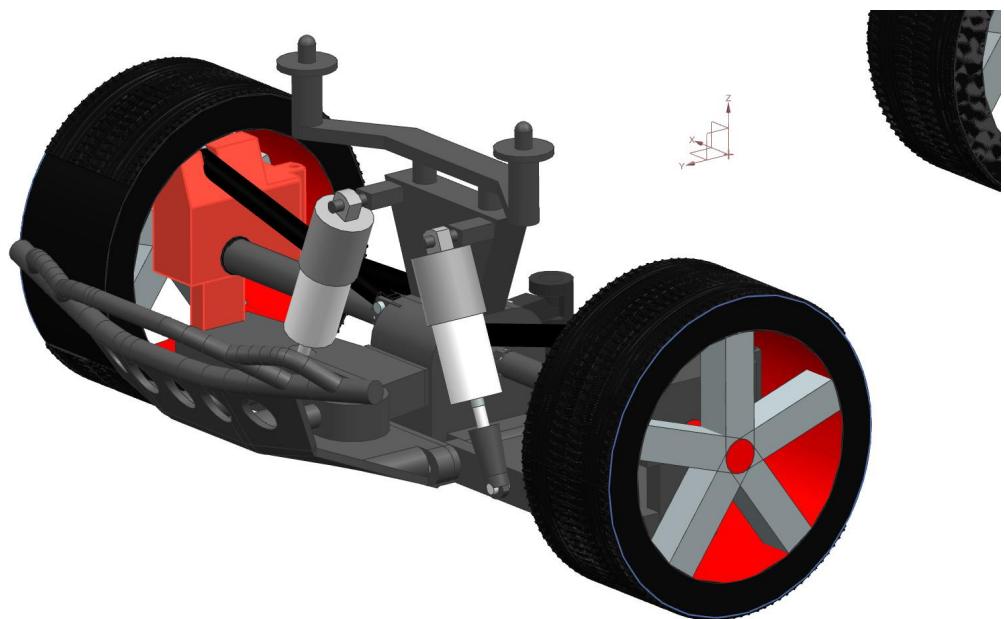
## Secondary Surface Modeling



# CAD Overview

## Solid Modeling: Suspension Subassemblies

- Select parts were modeled with Optimization in mind



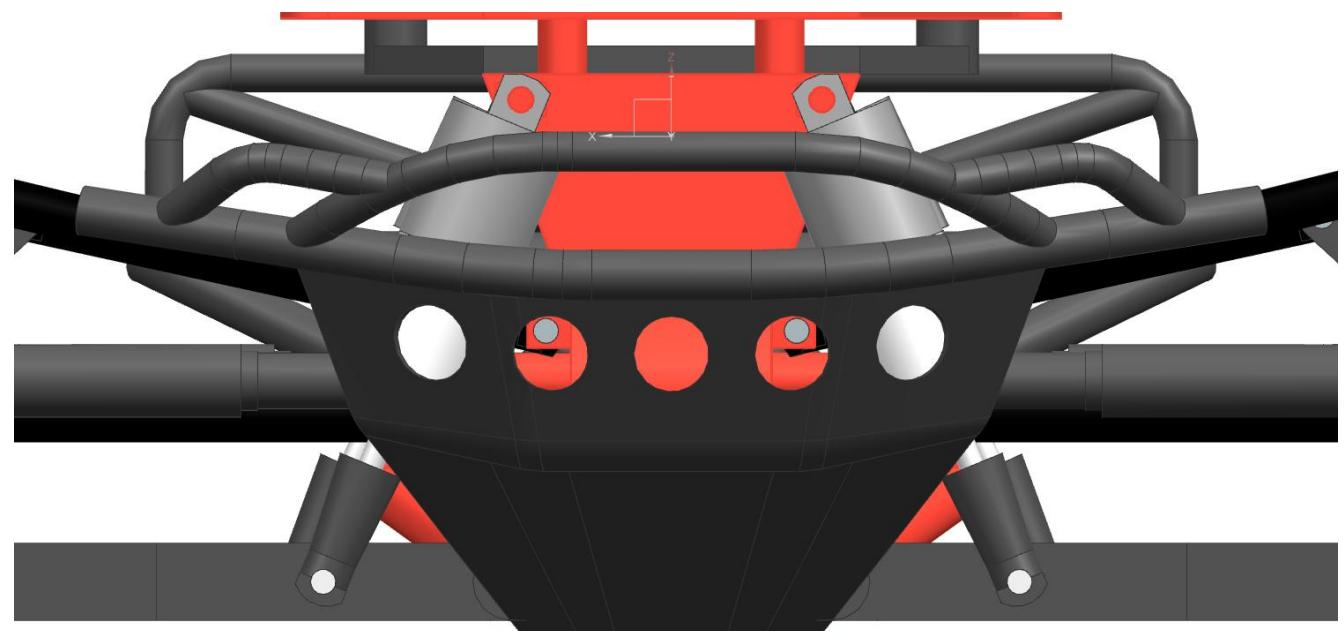
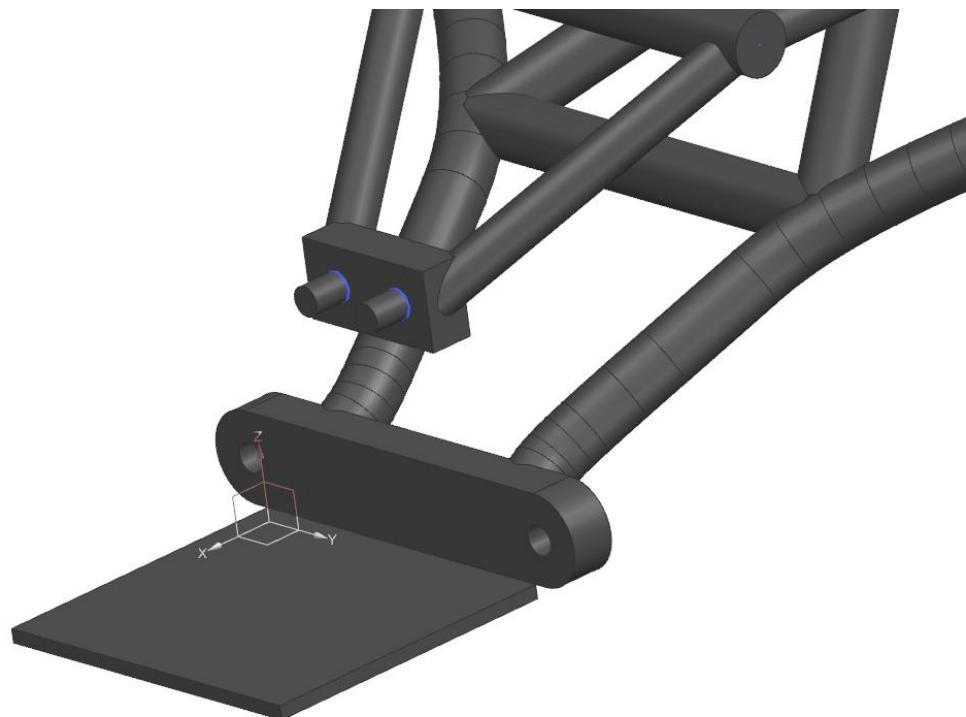
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# CAD Overview

## Solid Modeling: Suspension Re-Engineering

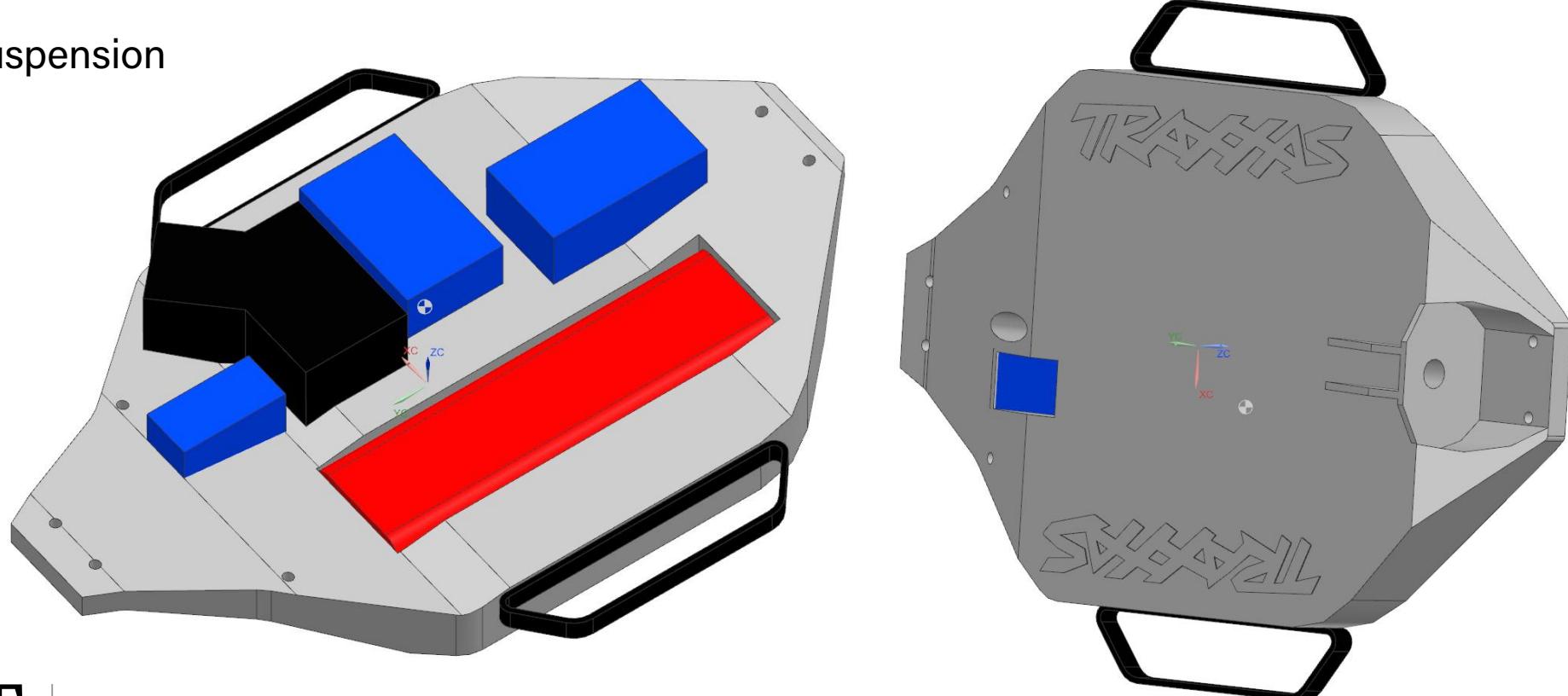
- Rear suspension had no upper supports
- Rear suspension location change



# CAD Overview

## Solid Modeling: Chassis

- Modeled with optimization in mind
- More time for finer details
- Integrate with front & rear suspension



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# CAD Overview

## Mass Properties

- Mostly ABS Plastic (high impact in Ansys)
  - Shell is Polycarbonate
- Yield Strength: 40 ksi
- Differs in CAD because of lack of optimized parts and design improvements

Part Description	Qty	Mass (g)	CAD Mass (g)	Difference	NOTES
Chassis	1	1165	1390	-225	Chassis left blank for optimization
Front Suspension	1	645	860	-215	Differs because topology optimization not considered/no cutouts, solid shocks, etc.
Rear Suspension	1	636	680	-44	^^See above
Shell	1	198	79	119	Optimized shell thickness
Total		2644	3009	-365	

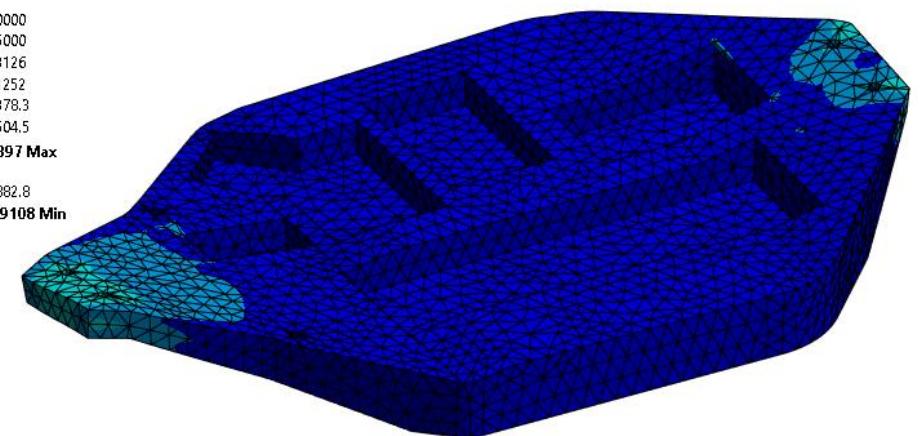
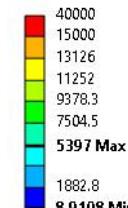
# *ANALYSIS AND OPTIMIZATION*

# *Analysis and Optimization*

## Chassis Torsion and Impact FEA

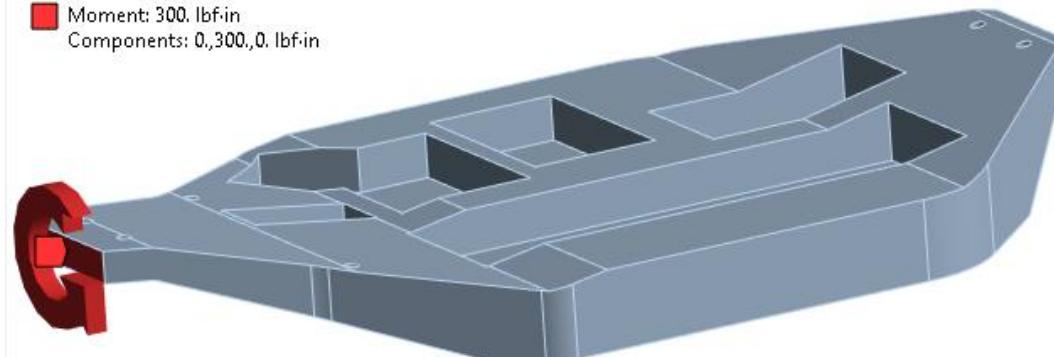
- Fixed back plane
- Cylindrical Supports at bolt holes
- Tetrahedral Mesh size: .25"
- Moment applied at front face
- Torsional stiffness provides basis for future assembly FEA and validates rigid constraints. Low deflection shows this is strong enough to assume this.

A: Static Structural  
Equivalent Stress  
Type: Equivalent (von-Mises) Stress  
Unit: psi  
Time: 1 s  
12/4/2024 1:47 AM



A: Static Structural  
Moment  
Time: 1. s  
12/4/2024 1:51 AM

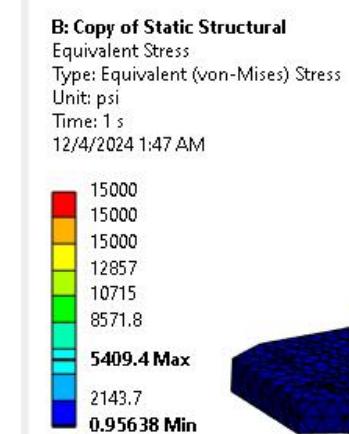
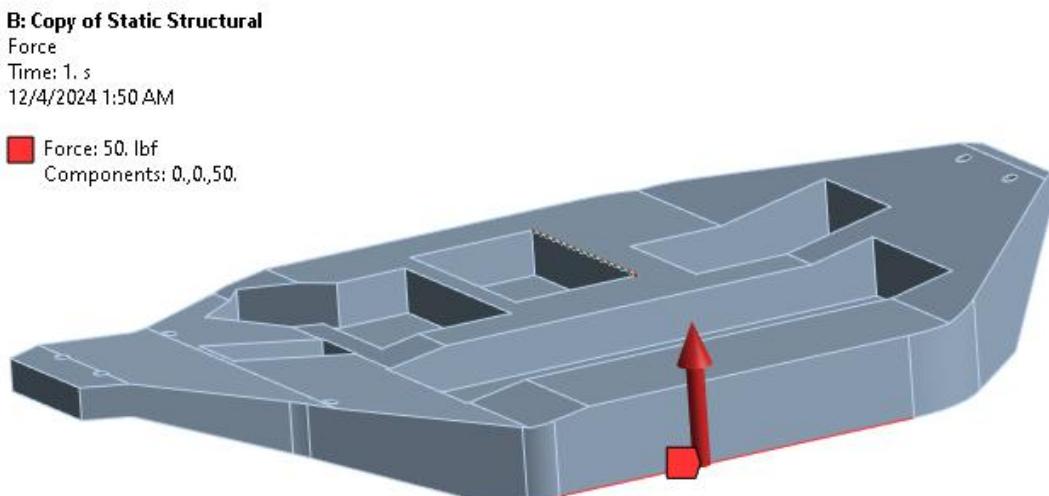
Moment: 300. lbf-in  
Components: 0,300.,0. lbf-in



# *Analysis and Optimization*

## Chassis Side Impact FEA

- Same setup as previous
- 50lbs on leftmost edge upward
- Stress is high but not close to 2.0 FOS



# *Analysis and Optimization*

## Chassis Front Impact FEA

- Same Fixed Geometry
- 500lbf applied at front face
- Low stress allows for much material removal

C: Copy of Copy of Static Structural

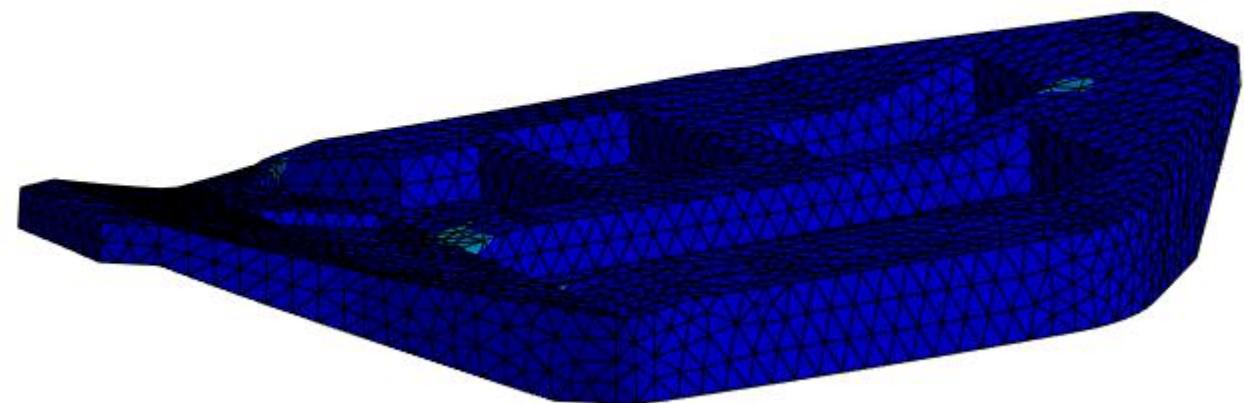
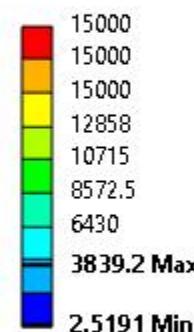
Equivalent Stress

Type: Equivalent (von-Mises) Stress

Unit: psi

Time: 1 s

12/4/2024 1:48 AM



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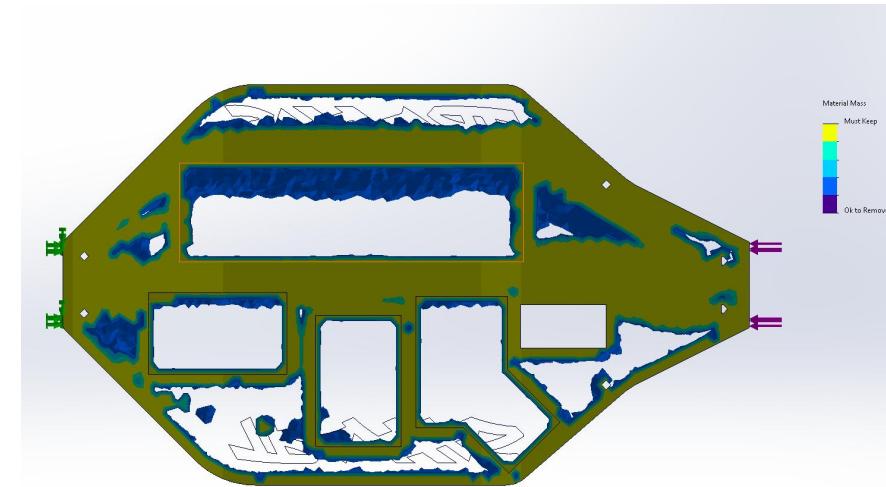
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# *Analysis and Optimization*

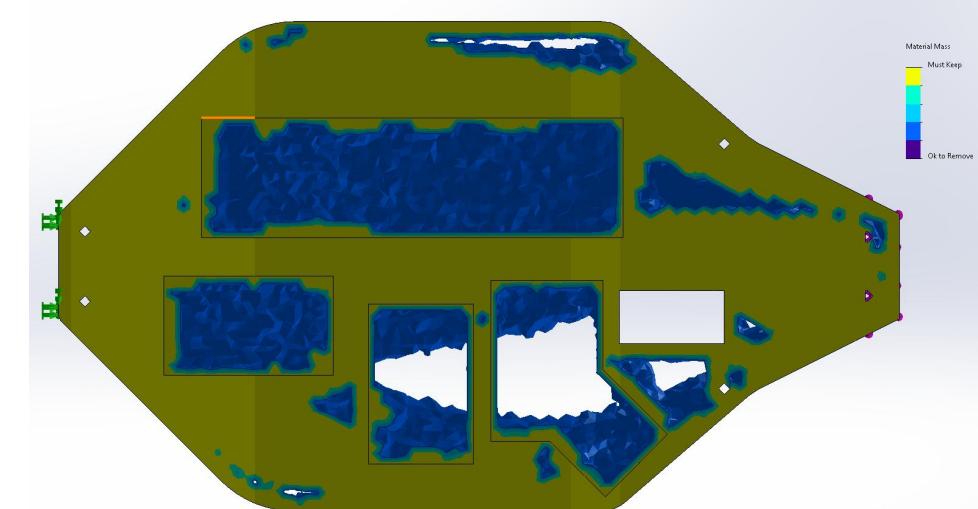
## Chassis Topology Optimization

- Kept outside geometry
- Fixed back face
- 20% weight reduction and 1.3 stiffness factor
- Force applied at front edge to simulate front impact, putting chassis in bending
- Significant amount of material can be removed

Study 1



Study 2



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# *Analysis and Optimization*

## Drivetrain FEA Torque Calcs

Motor Data:

Titan 12T

Max Torque Output: 1.5-2Nm

Gear Ratio: 3.57:1

Pressure Angle: 20 deg.

Torque at Gear: 63.216 in-lbs

Equivalent Force Tangent = 75.708 lbs

Equivalent Force at Pressure angle: 208.005  
lbs

DRIVE MOTOR

Titan 12T

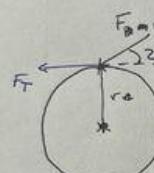
Torque output = 200 Nm

ODI Input Pinion =  $\frac{48T}{14}$

~~Ratio~~ =  $\frac{56T}{16T} = 3.57:1$  ~~T<sub>b</sub> = 7.143 Nm~~

$\frac{n_G}{T_b} = \frac{n_P}{T_P} \Rightarrow \frac{n_G}{n_P} = \frac{T_b}{T_P} \Rightarrow \frac{n_G}{n_P} T_P = T_b$

$T_b = 7.143 \text{ Nm}$  Pressure angle 20°  
 $= 5.268 \text{ ft-lbs} = 63.216 \text{ in-lbs}$



$r = 1.67 / 2 = .835 \text{ in} \Rightarrow$

$F_T = \frac{T_b}{r} = \frac{63.216 \text{ in-lbs}}{.835 \text{ in}} = 75.7078 \text{ lbs}$

~~Eqn~~  $\frac{F_T}{F_{Pa}} = \tan(20^\circ) \Rightarrow F_{Pa} = \frac{F_T}{\tan(20^\circ)}$



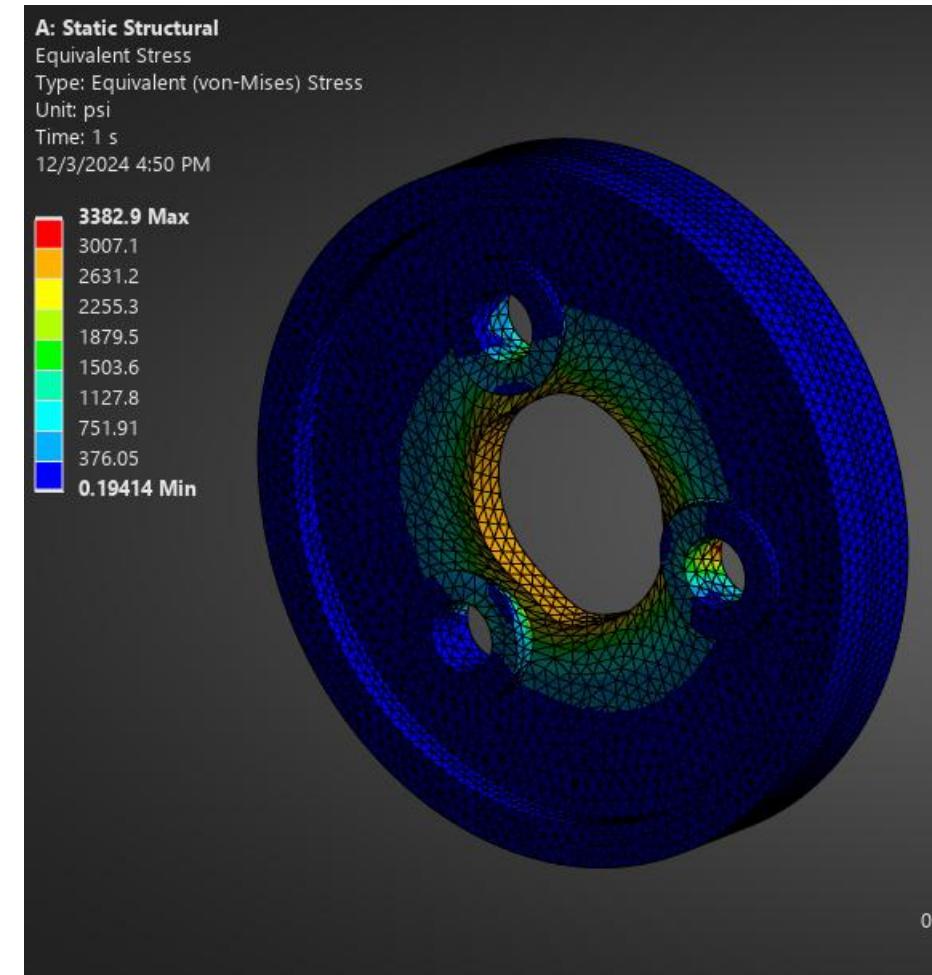
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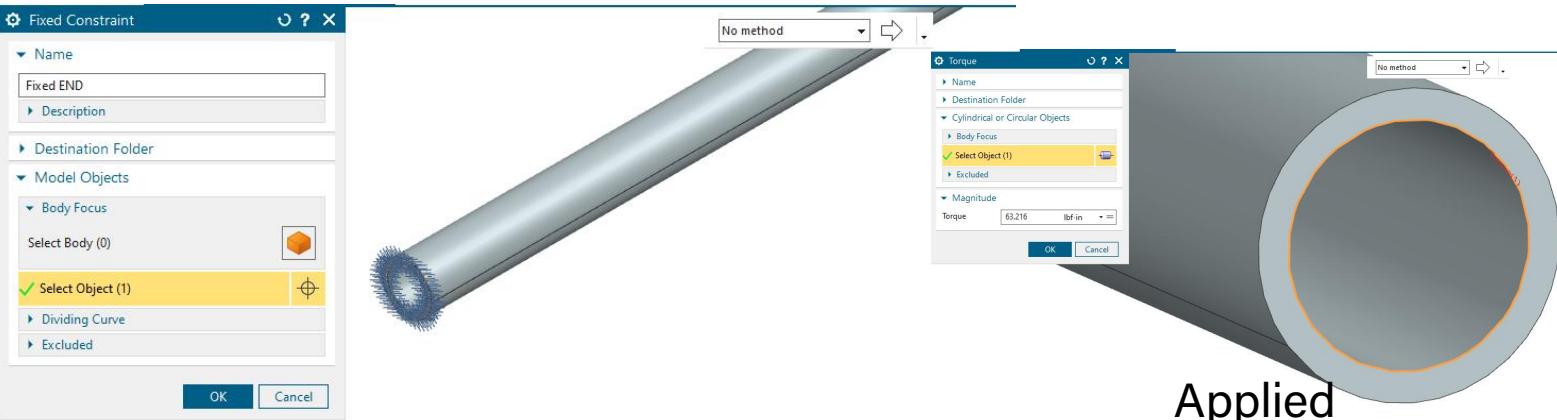
# *Analysis and Optimization*

## Drive Gear FEA

- Tetrahedral Mesh (size .035 in)
- ABS plastic (high impact)
- Torque applied from calculation: 63.2 in-lbs at center cylinder
- Fixed cylindrical supports at bolts and at edge
- Material Yield: 36 ksi

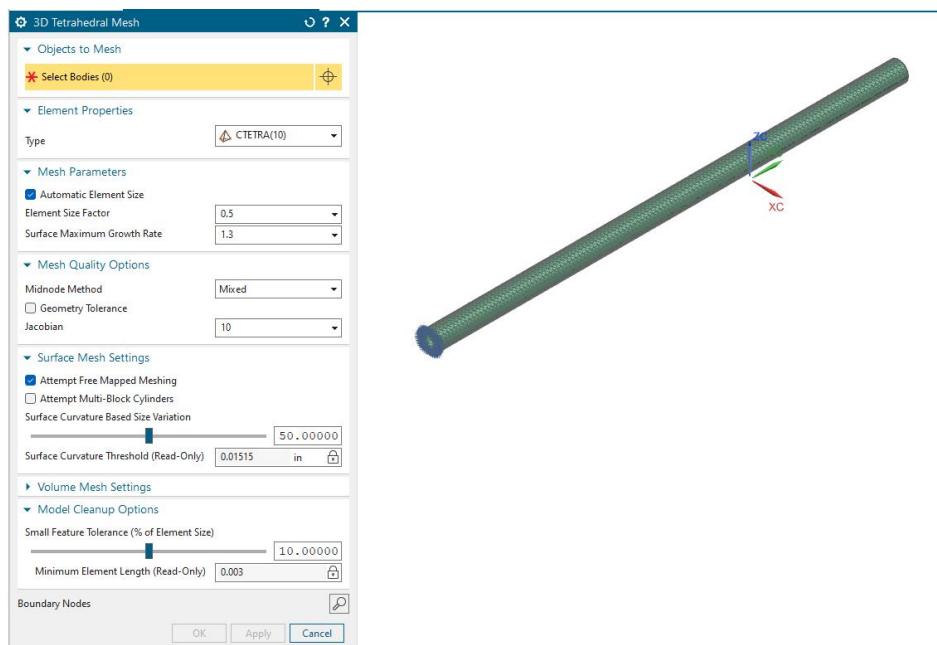


# Analysis and Optimization

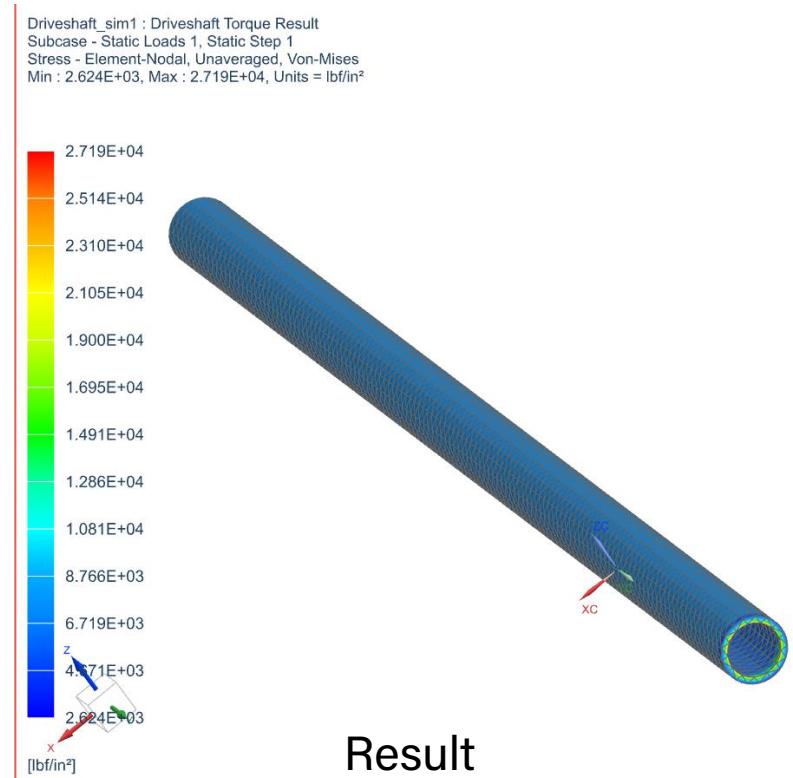


Fixed Constraint

Applied  
Torque



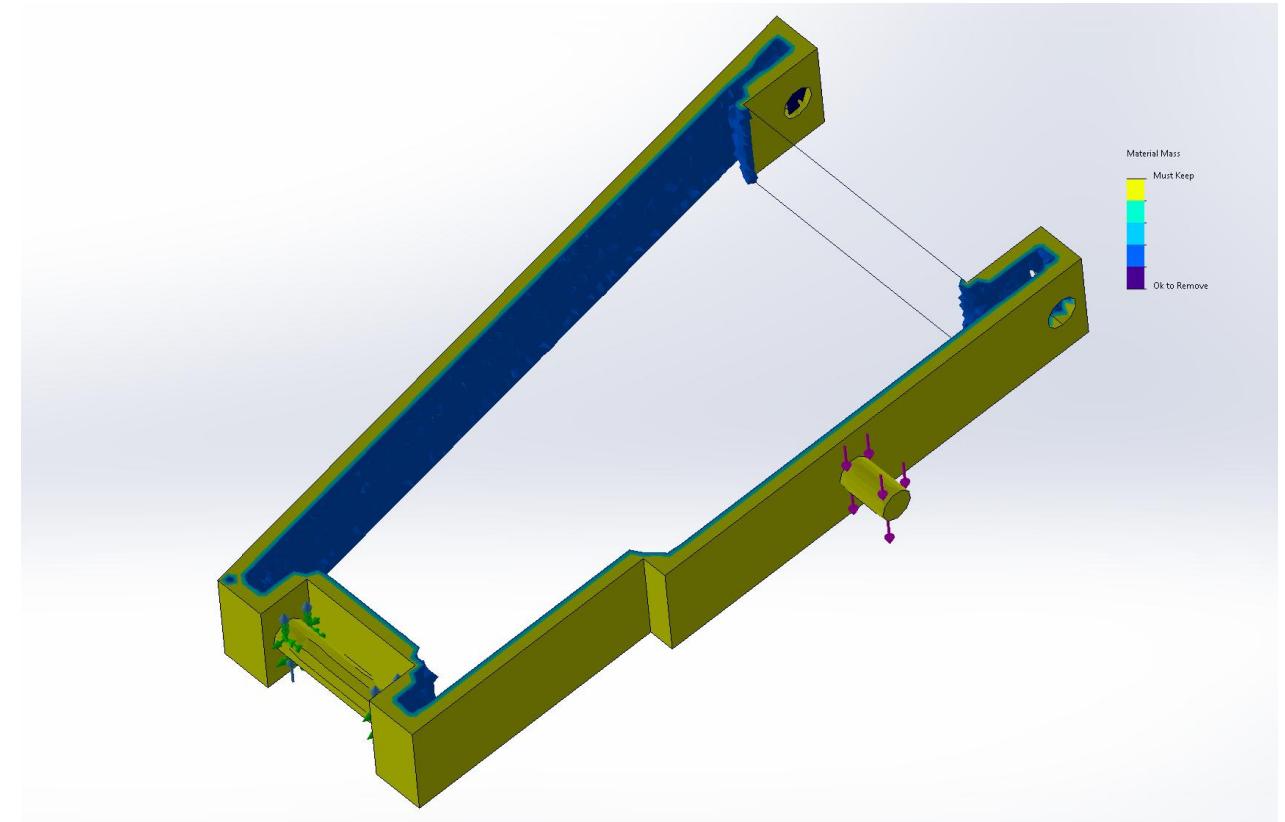
3D Tet  
Mesh



# *Analysis and Optimization*

## Bottom Support Topology Optimization

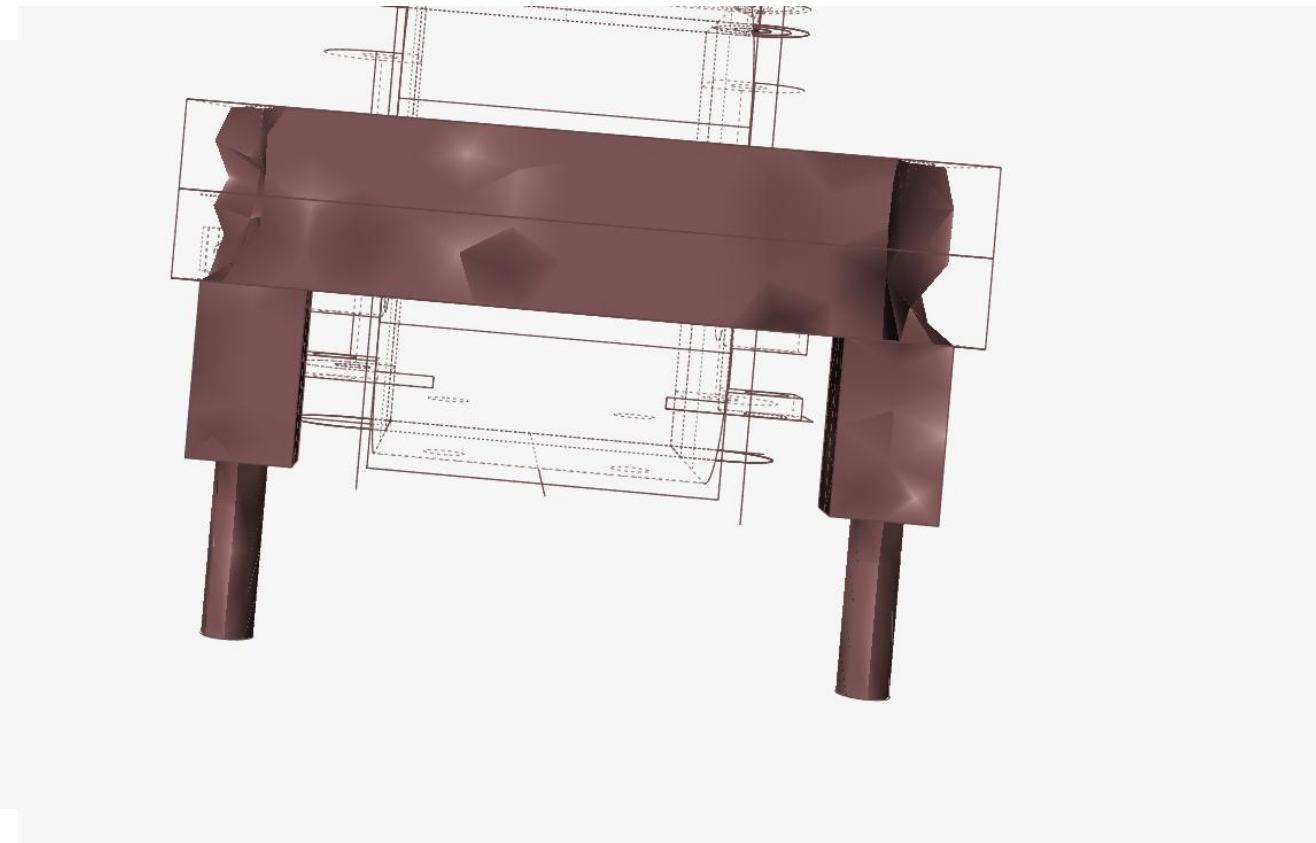
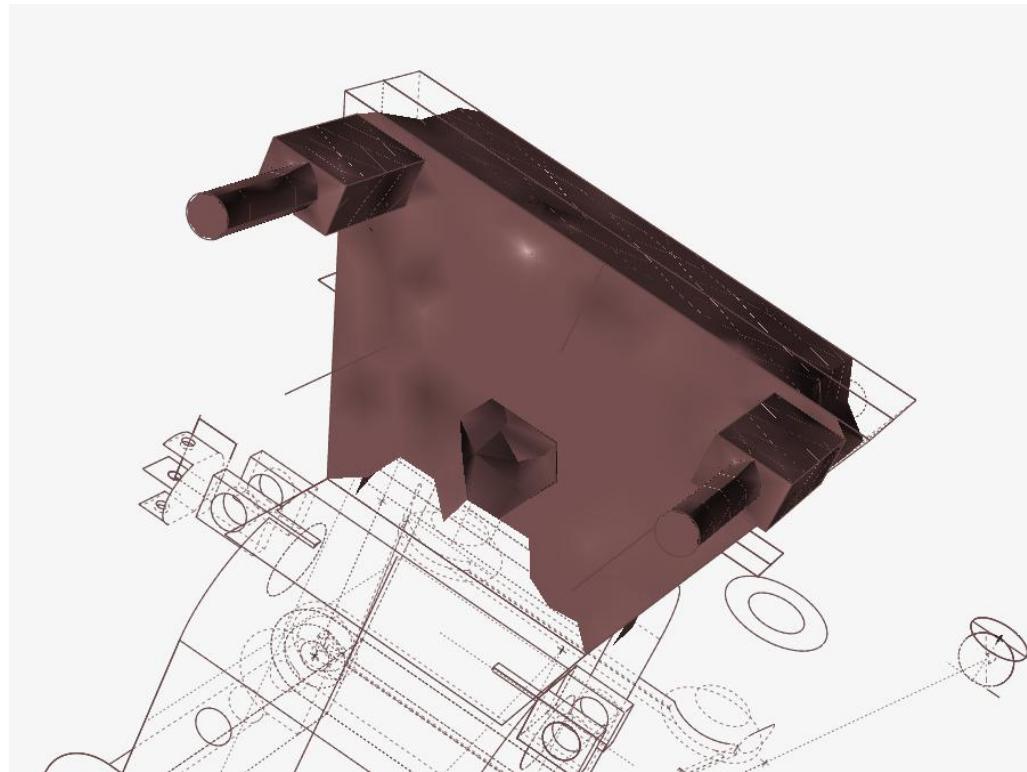
- Cylindrical Supports at bolt holes and bearing
- Keep Geometry outline
- Forces applied at outer cylinder and shock mount point
- 20% weight reduction and 1.2 deflection factor
- Would need to keep material in center for additional stiffness



# *Analysis and Optimization*

## Suspension Center Piece Topology Optimization

- Fixed geometry
- 100 lb applied at 30 degrees



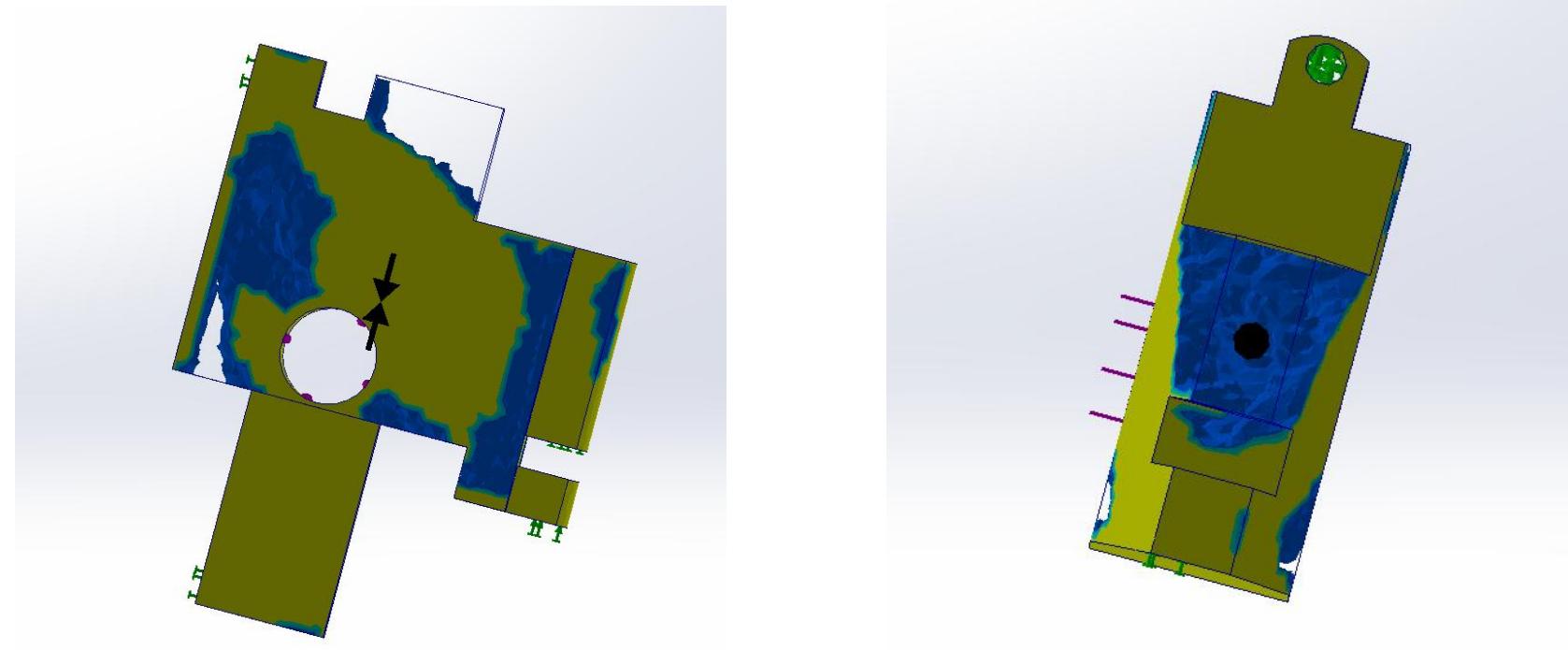
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# *Analysis and Optimization*

## Wheel Attachment Topology Optimization

- Manufacturing constraints on holes for bolts and wheel
- Fixed geometry at outer boundaries
- Force Applied at the Wheel Center Inboard
- 10% weight reduction and 1.1 deflection factor

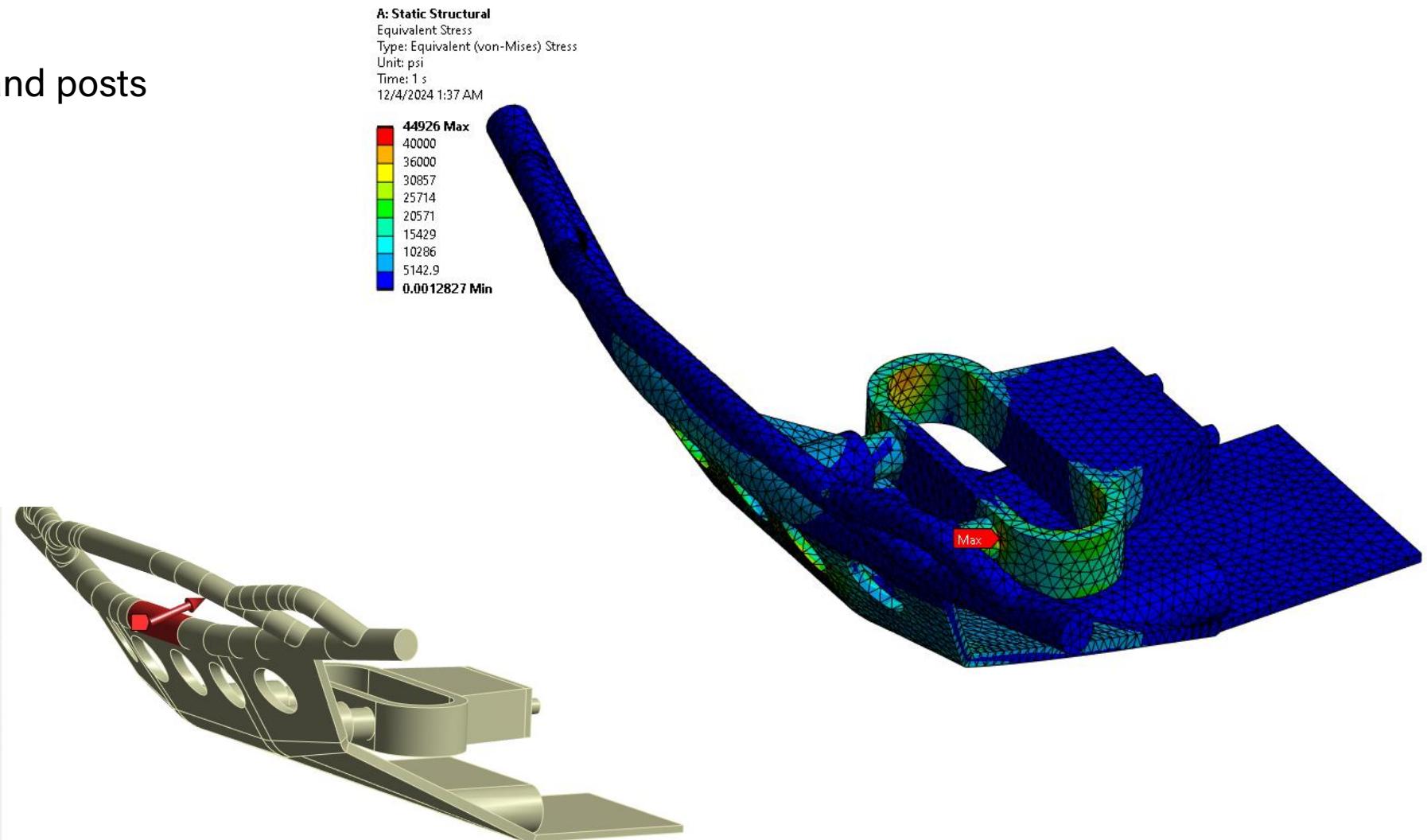


# *Analysis and Optimization*

## Front Bumper FEA

- Cylindrical Supports at bolts and posts
- Force applied as shown
- Tetrahedral Mesh Size: .1"

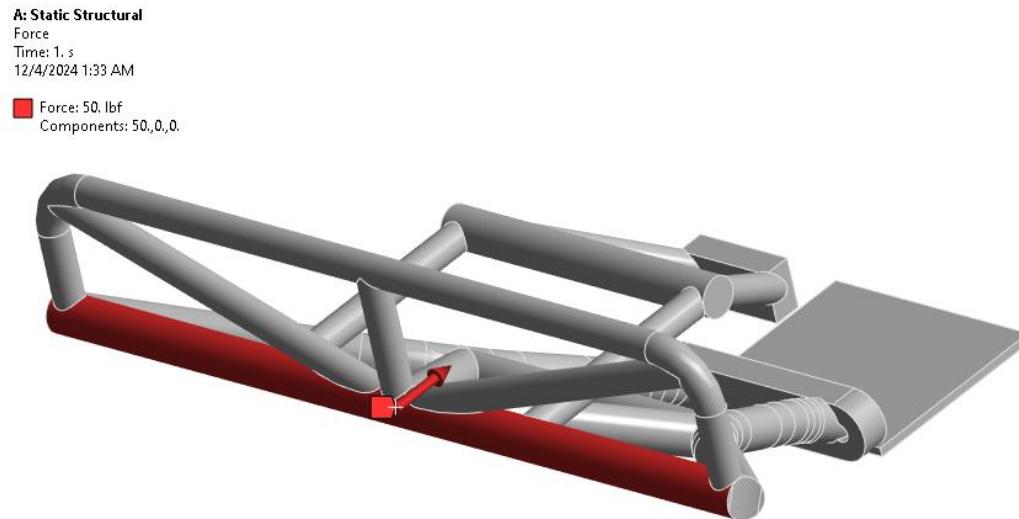
Details of "Force"	
<b>Scope</b>	
Scoping Method	Geometry Selection
Geometry	1 Face
<b>Definition</b>	
Type	Force
Define By	Components
Applied By	Surface Effect
Coordinate System	Global Coordinate System
<input type="checkbox"/> X Component	0. lbf (ramped)
<input type="checkbox"/> Y Component	-150. lbf (ramped)
<input type="checkbox"/> Z Component	50. lbf (ramped)
Suppressed	No



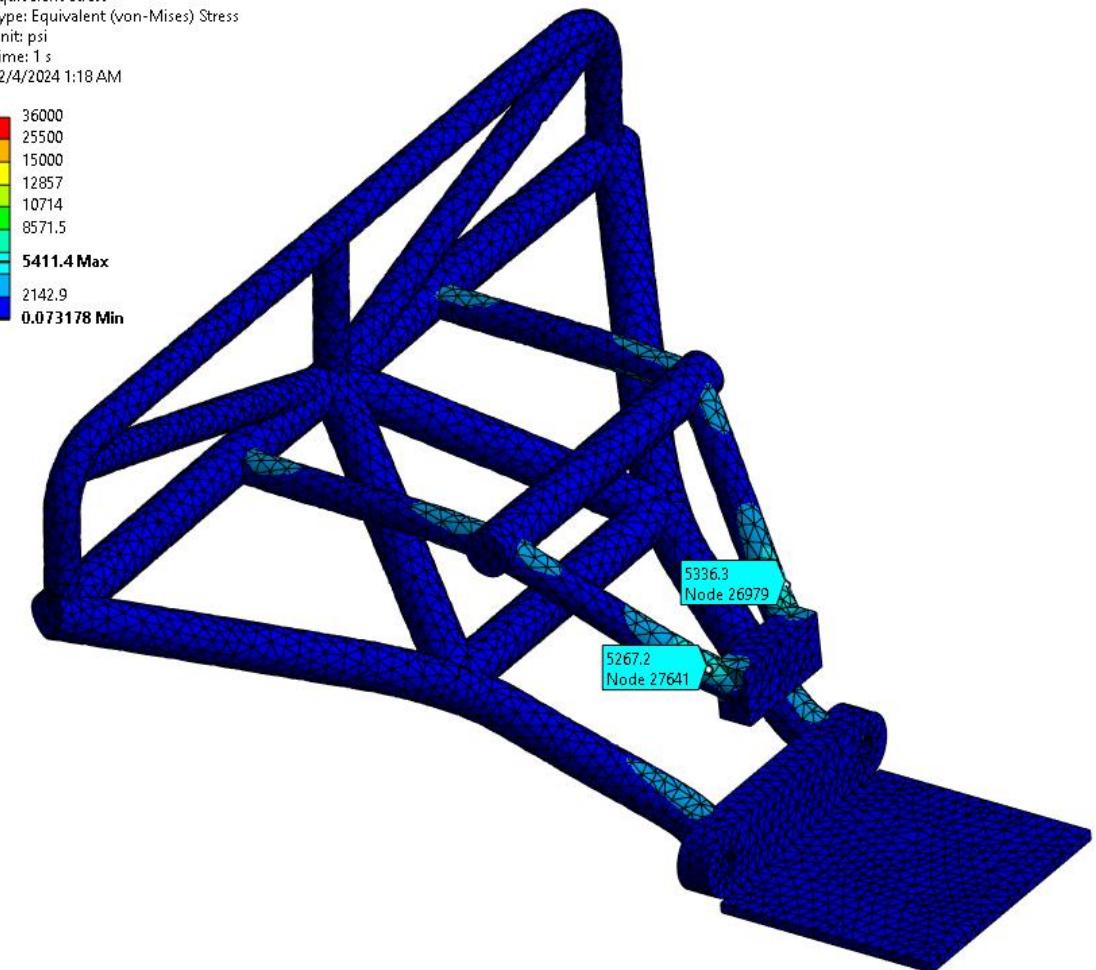
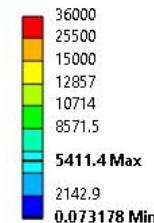
# *Analysis and Optimization*

## Rear Bumper FEA

- Cylindrical Supports at bolt holes, all fixed
- Tetrahedral Mesh size: .1"
- Force applied: 50 lbf



A: Static Structural  
Equivalent Stress  
Type: Equivalent (von-Mises) Stress  
Unit: psi  
Time: 1 s  
12/4/2024 1:18 AM



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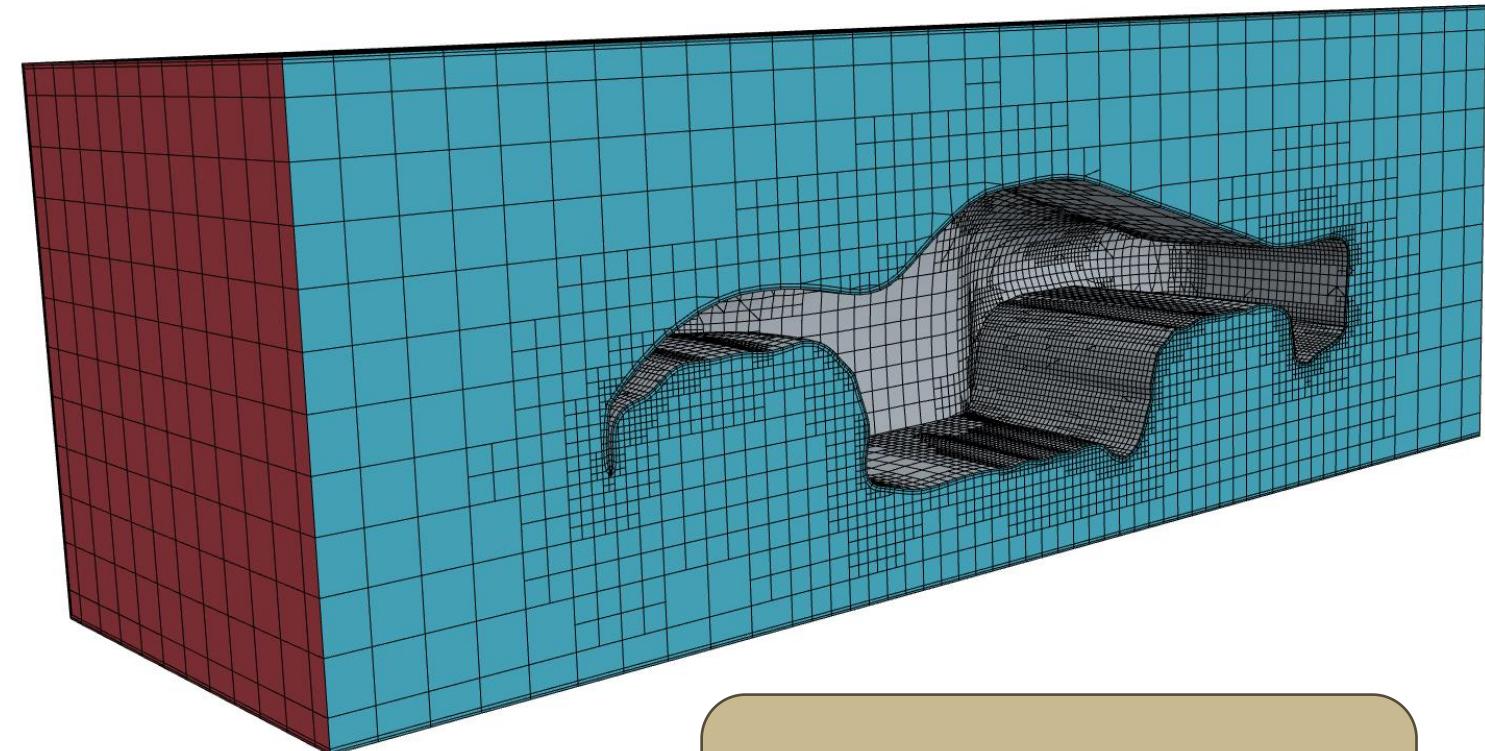
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# *CFD*

# CFD

## Shell CFD

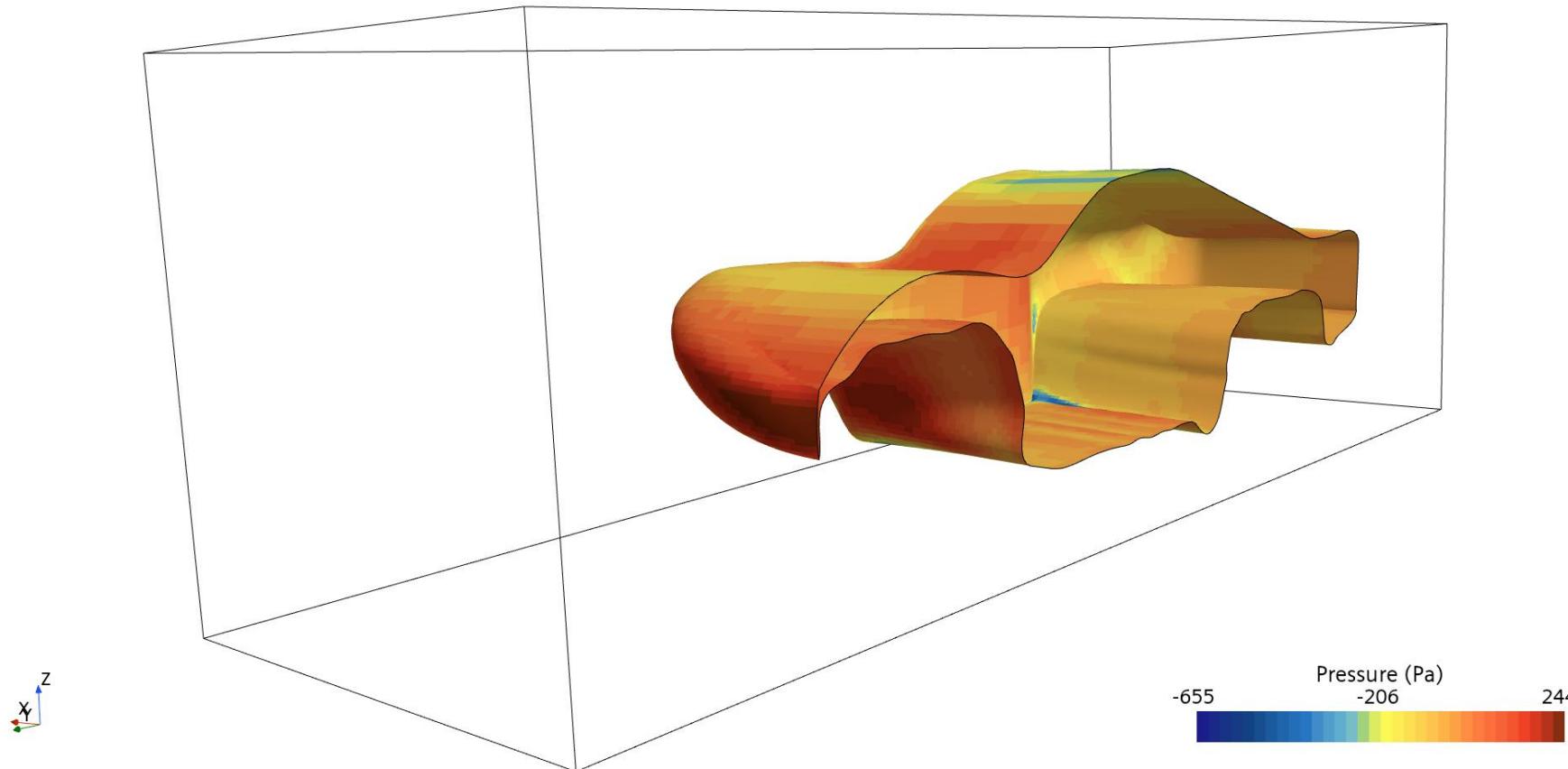
- Mock wind tunnel created in CAD
- Meshers: Surface Remesher, Trimmed Cell Mesher, Prism Layer Mesher
- Element Size: 0.025m
- Prism Layer Total Thickness: 10% (of base size)
- Physics Model Selection: Gas, Segregated Flow, Ideal Gas, Segregated Fluid Temperature, Steady, Turbulent, K-Epsilon, Turbulence



**Test Cases: 36 mph, 45 mph,  
70 mph, 100 mph**

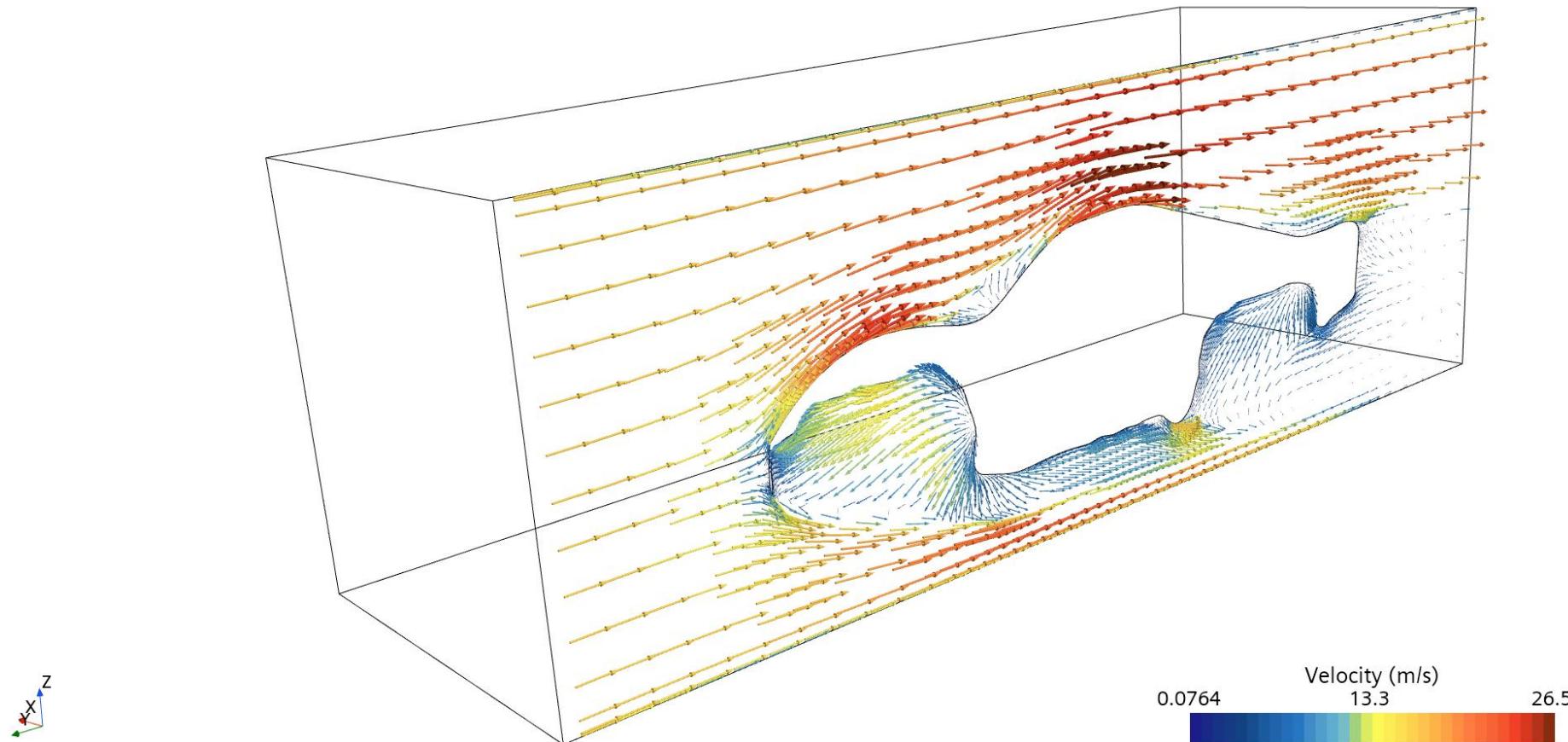
*CFD*

## 36 mph - Scalar Scene



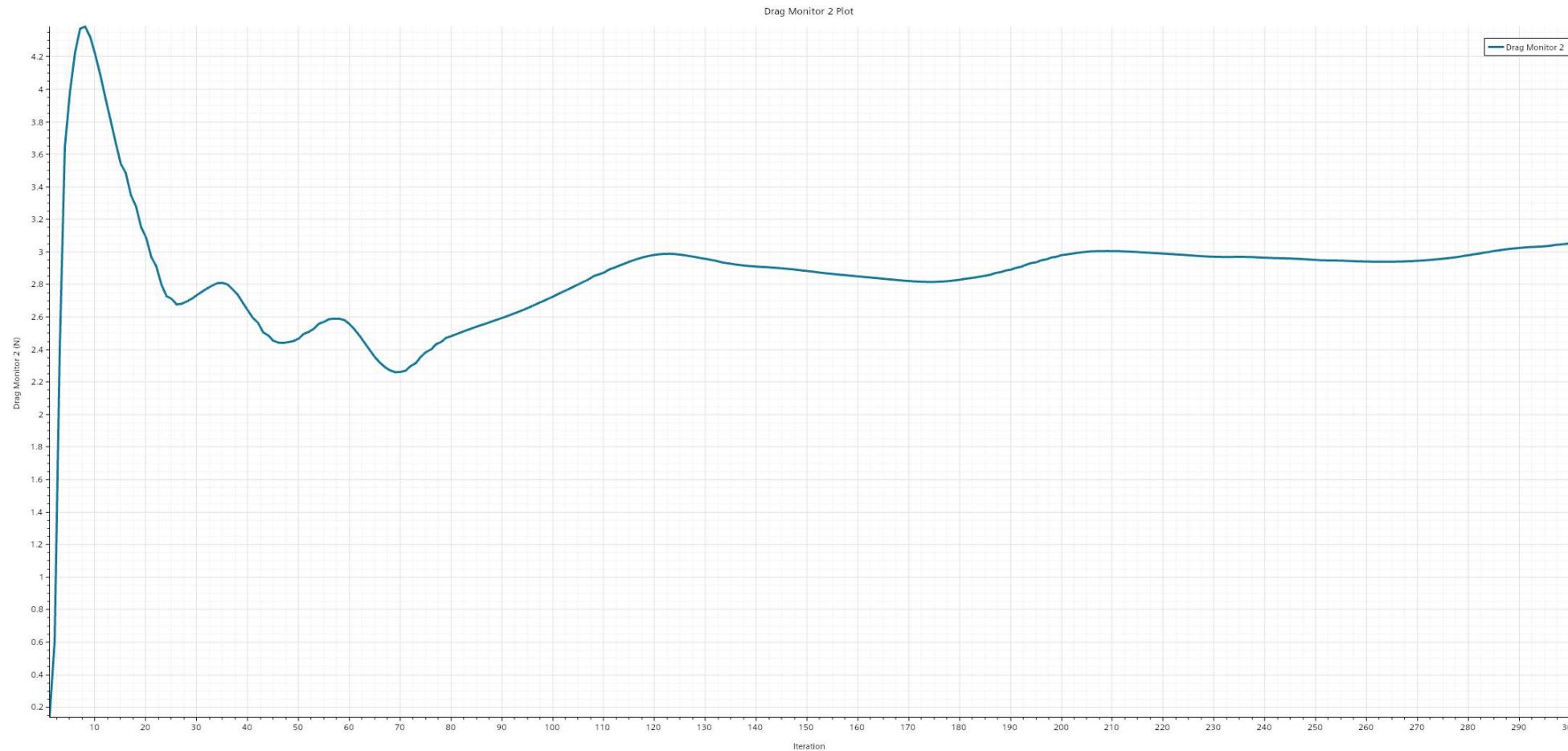
*CFD*

## 36 mph - Vector Scene



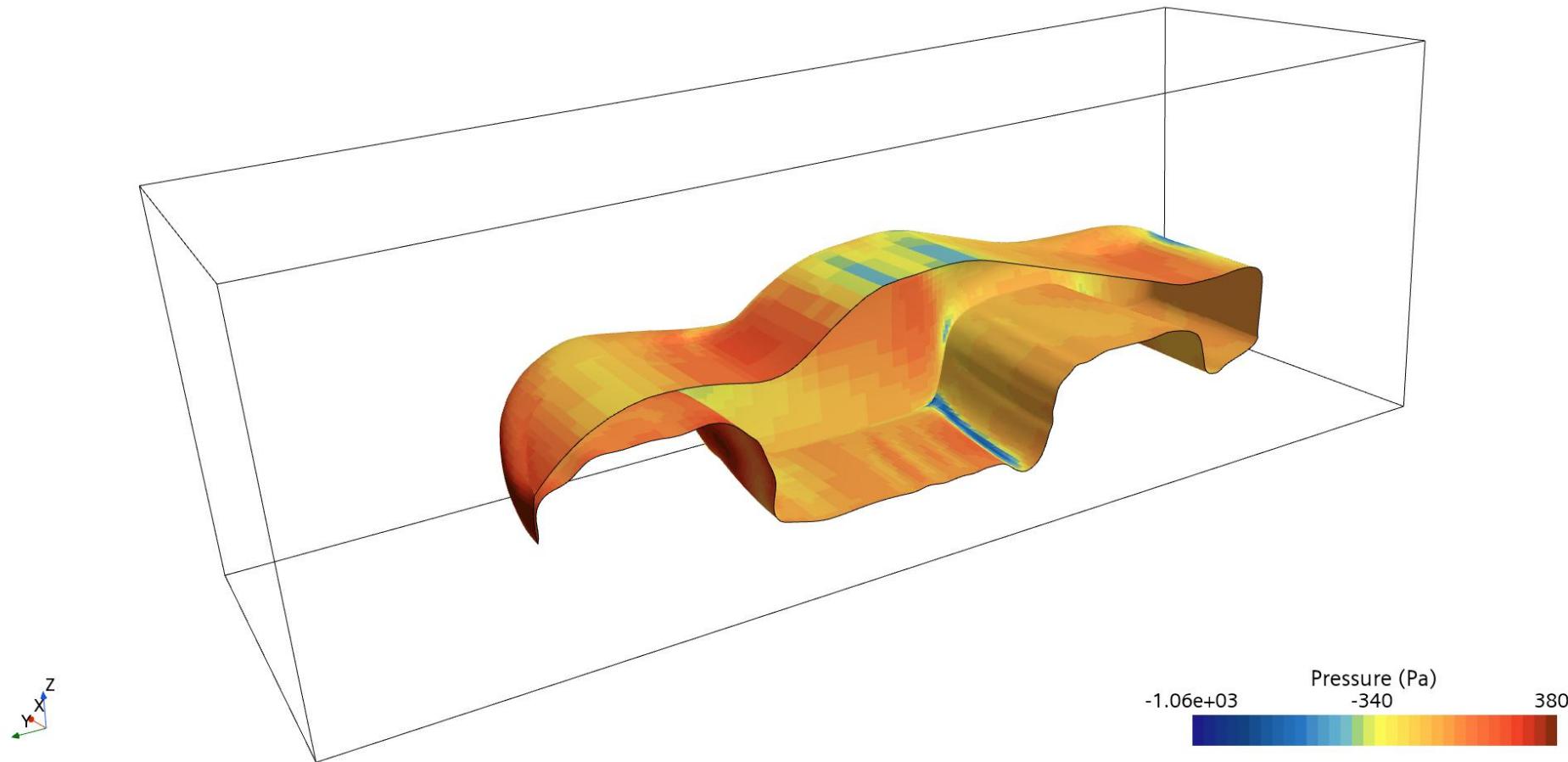
**CFD**

## 36 mph - Drag Force Plot



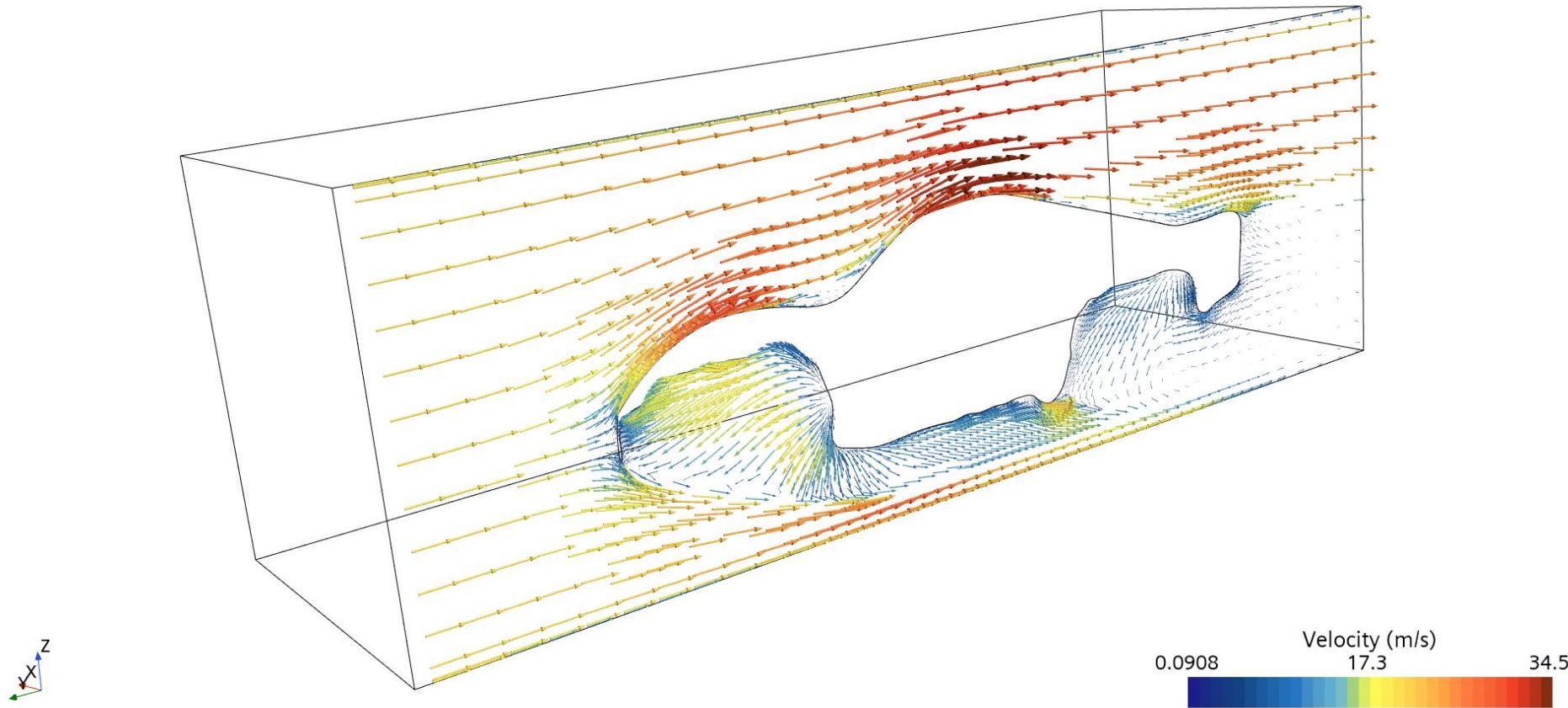
*CFD*

## 45 mph - Scalar Scene



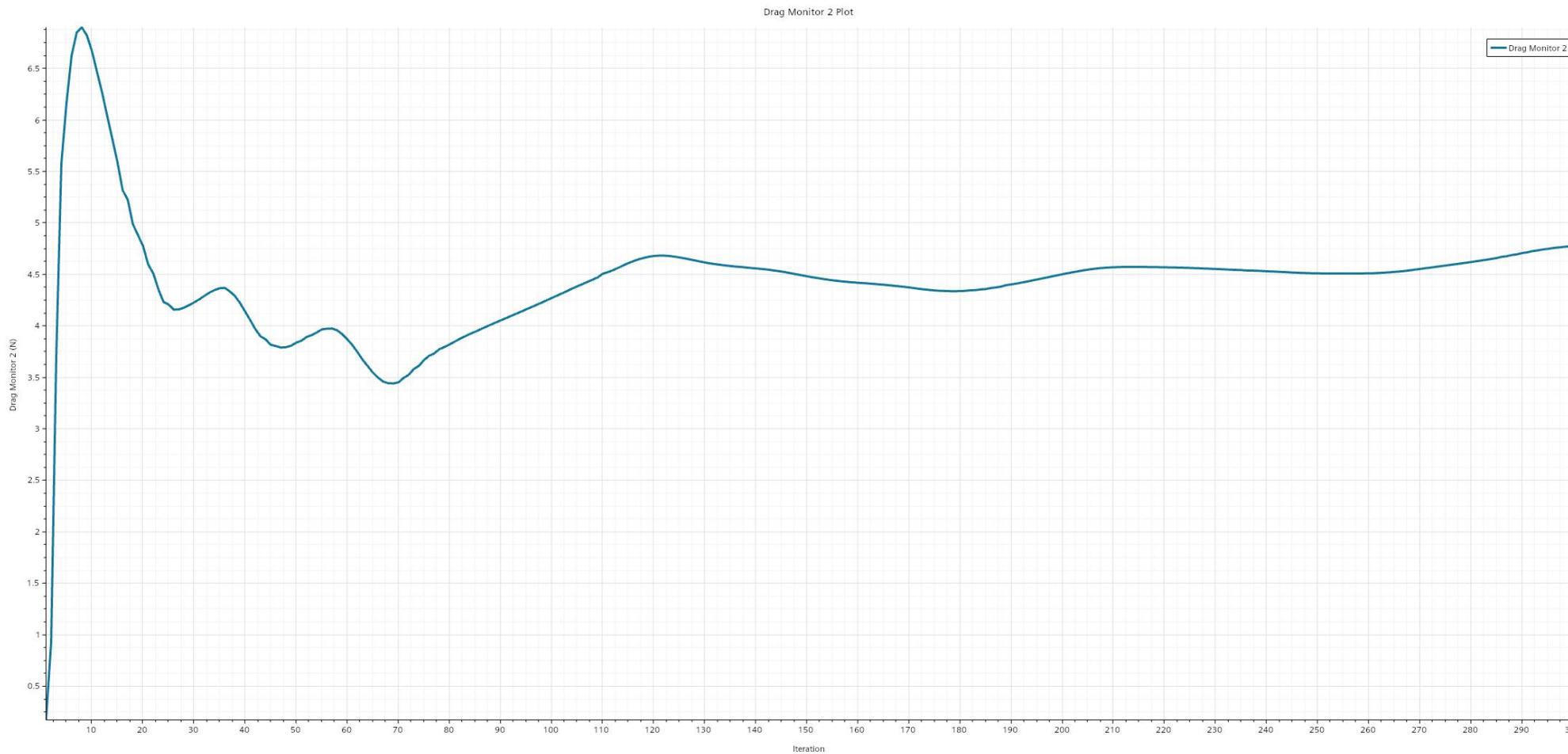
*CFD*

## 45 mph - Vector Scene



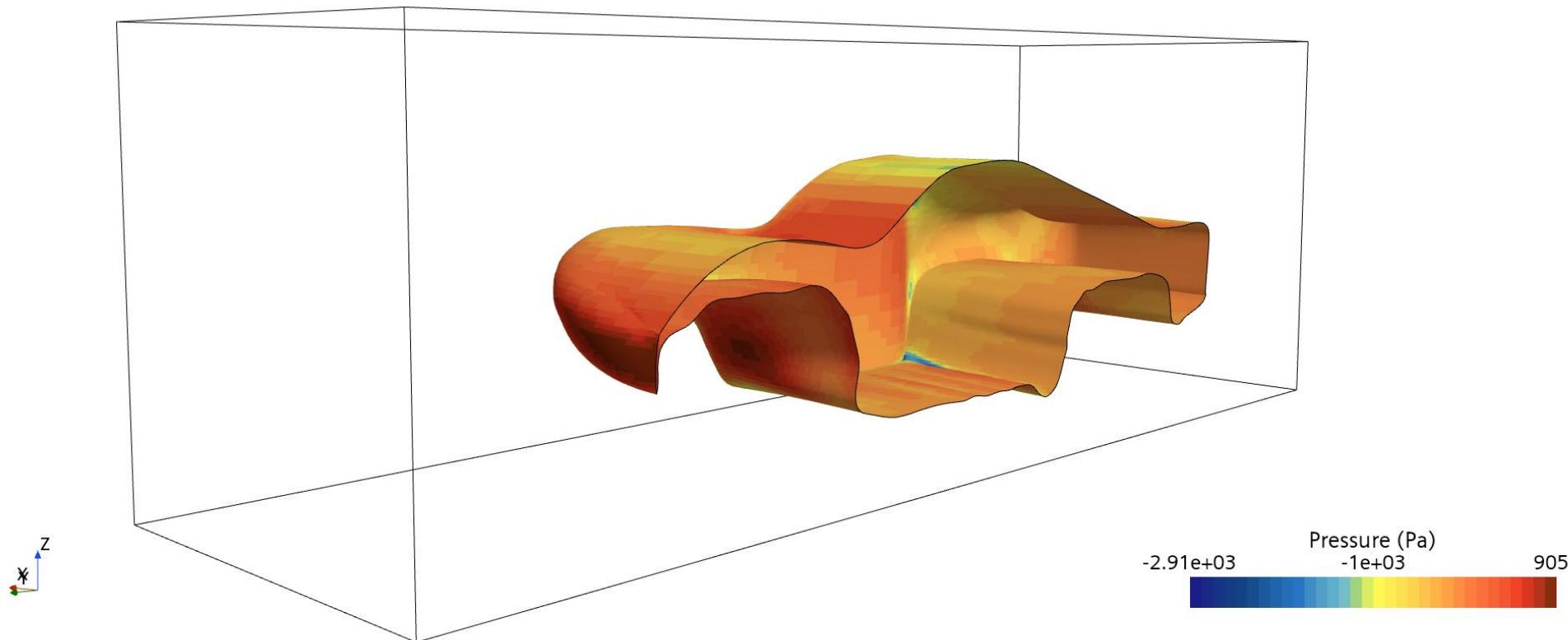
*CFD*

## 45 mph - Drag Force Plot



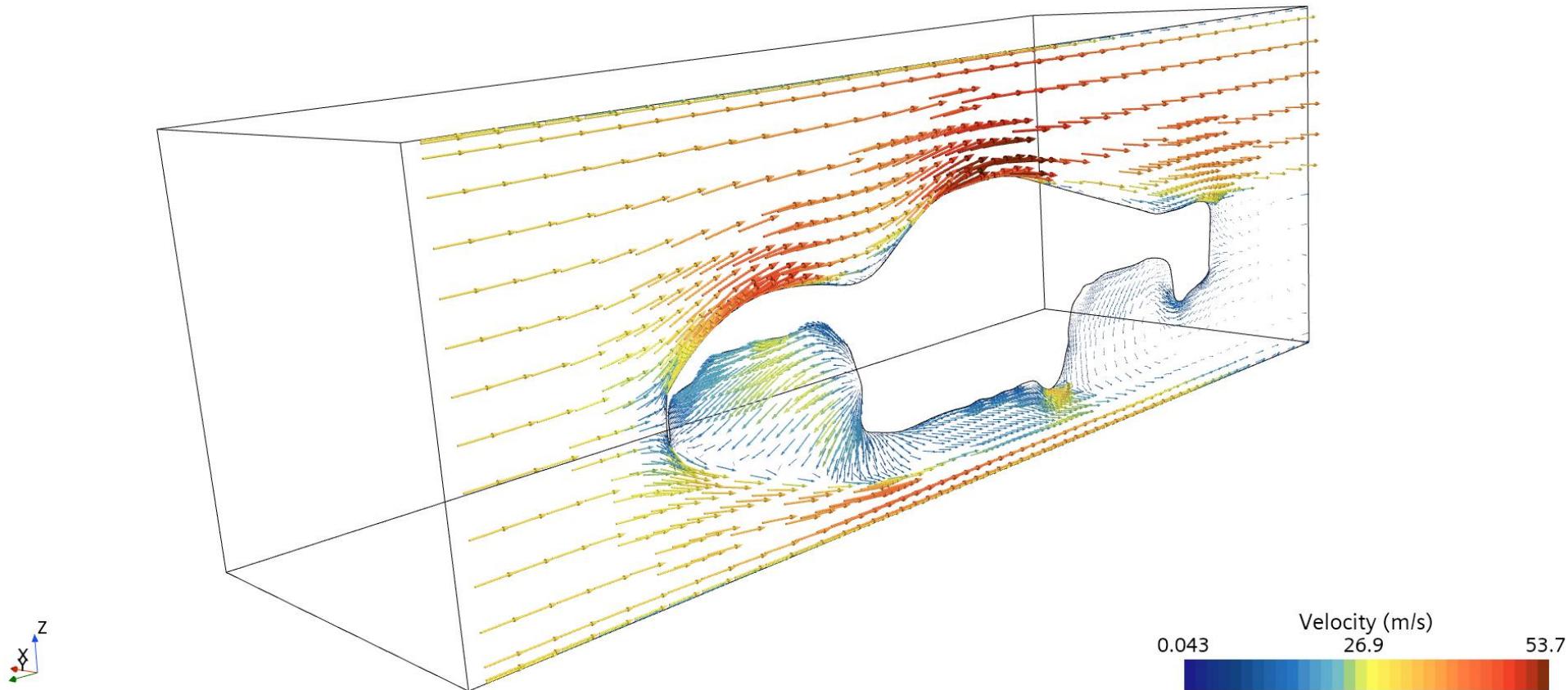
*CFD*

## 70 mph - Scalar Scene



*CFD*

## 70 mph - Vector Scene

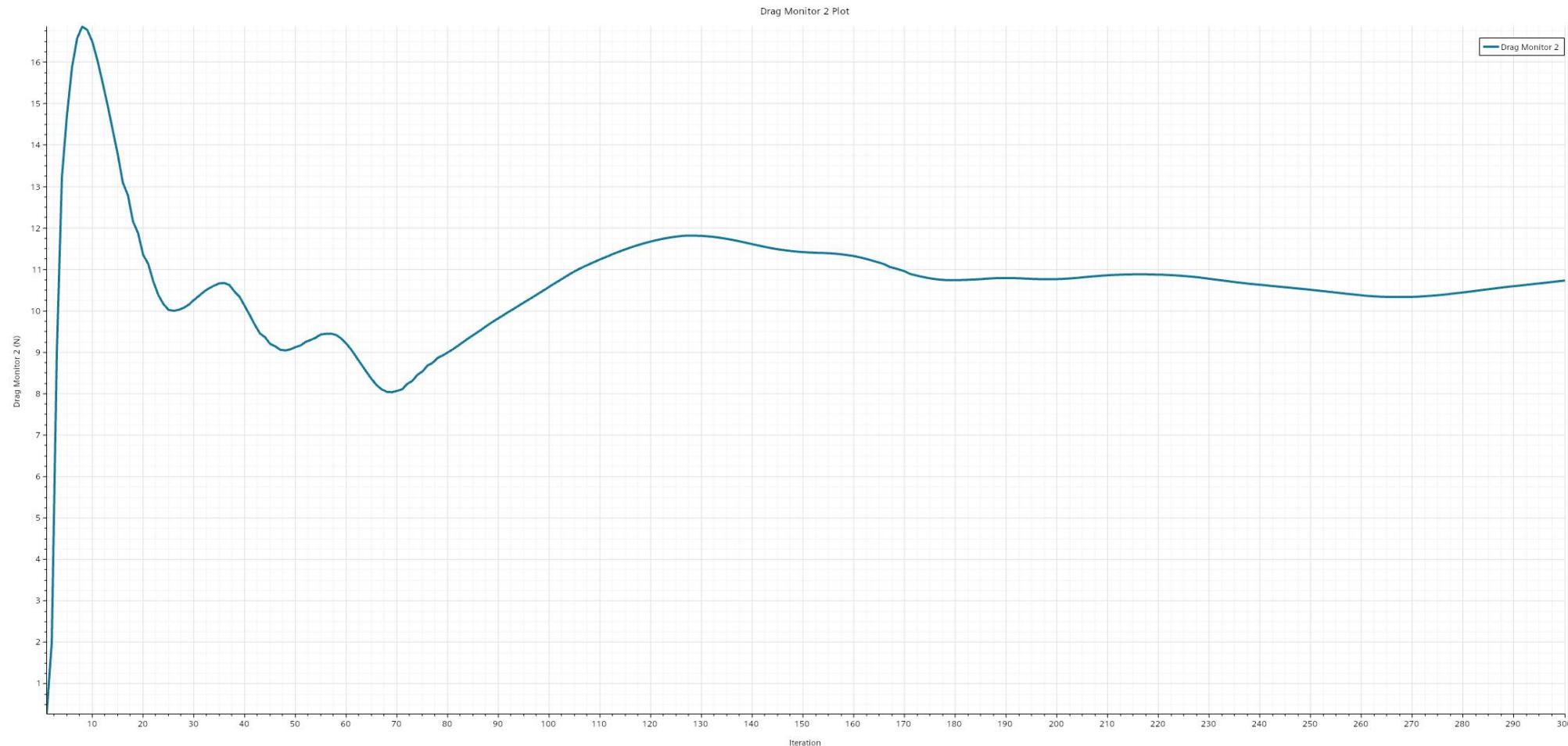


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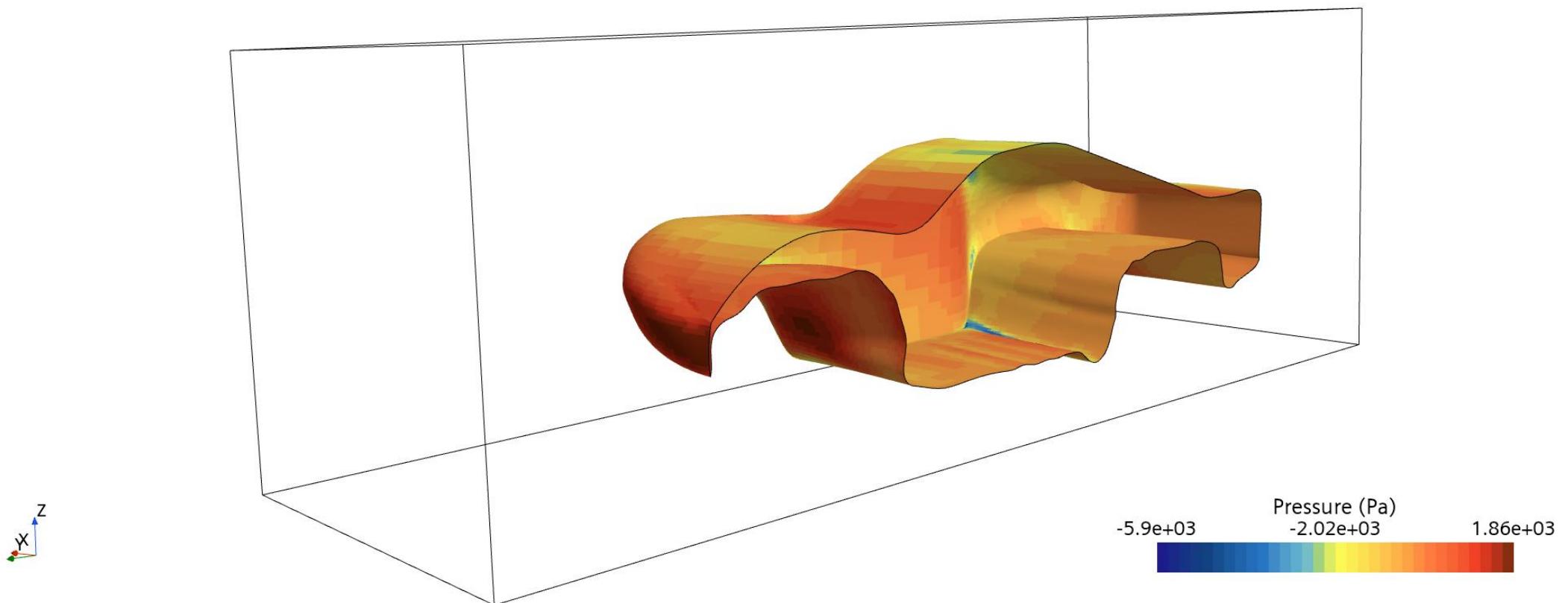
*CFD*

## 70 mph - Drag Force Plot



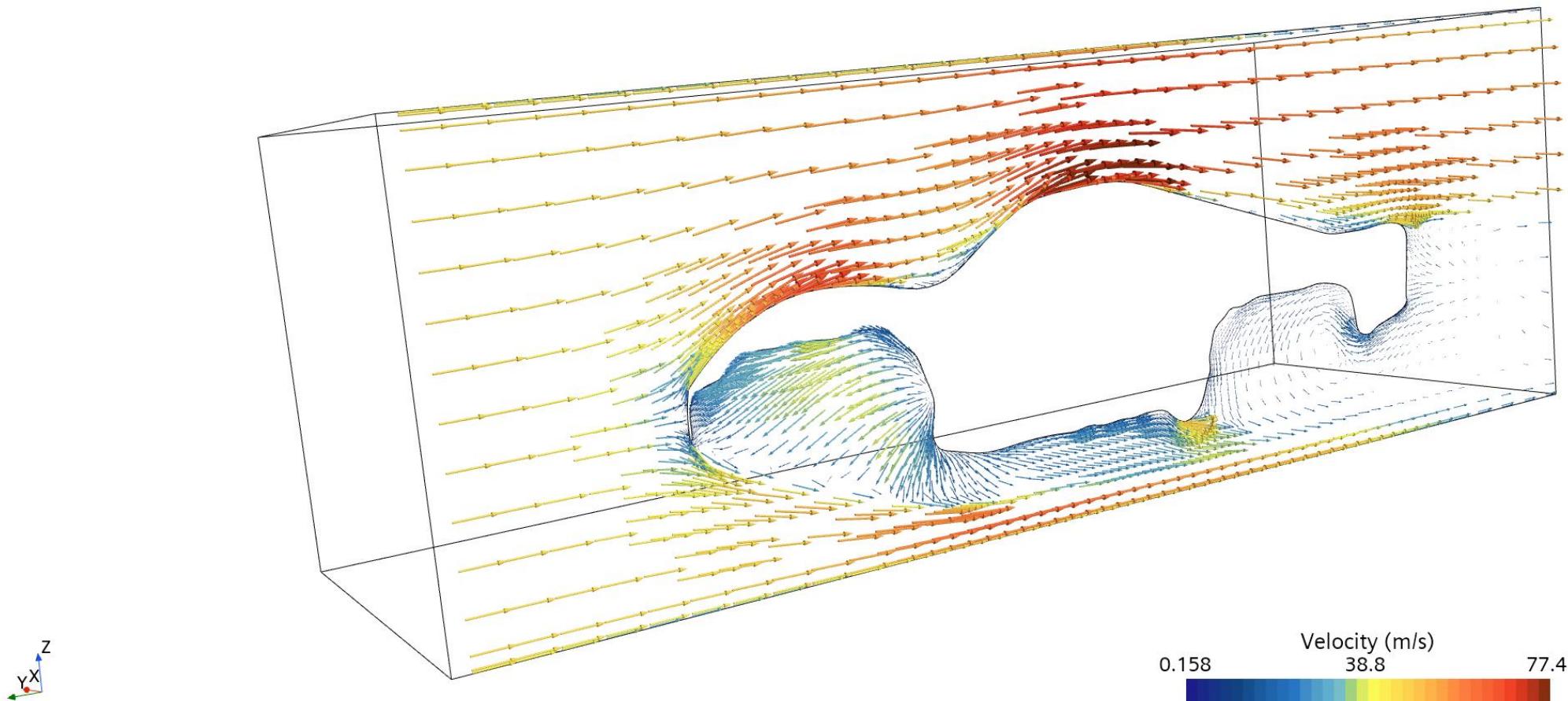
*CFD*

## 100 mph - Scalar Scene



*CFD*

## 100 mph - Vector Scene

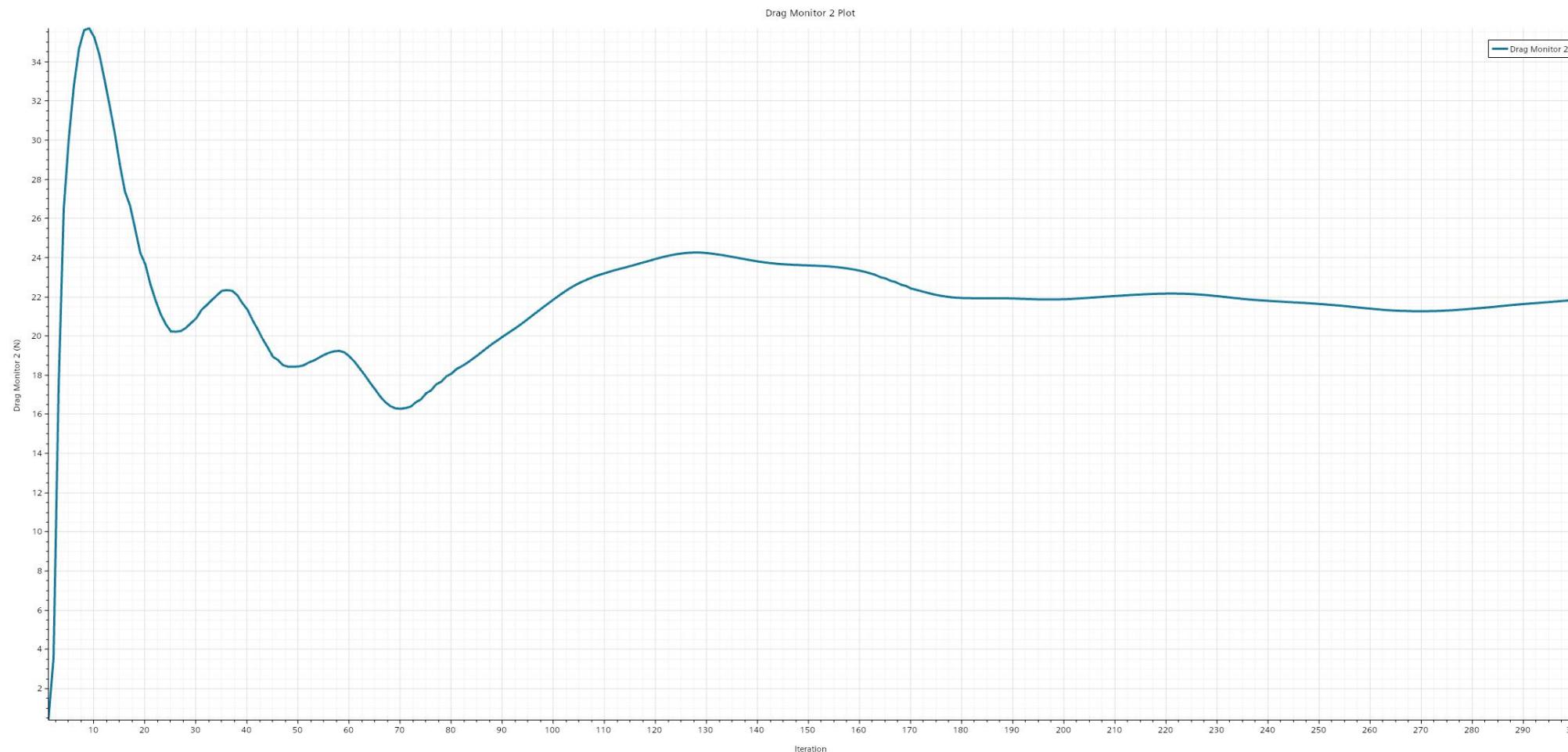


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**CFD**

## 100 mph - Drag Force Plot



## Drag Force at 300<sup>th</sup> Iteration

- Drag Increases as speed goes up in expected manner
- Therefore, methodology conducted for CFD is valid and these results can be used to draw some conclusions

Case	Force (N)
36 mph	3 N
45 mph	4.75 N
70 mph	10.8 N
100 mph	22 N



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# VISUALIZATION

# Visualization

## Full Assembly Visualization



# Visualization

## Full Assembly Visualization



# *Visualization*

## Full Assembly Visualization



# Visualization

## Full Assembly Visualization



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# *Part Visualization*

## Suspension Assembly Visualization



# *Part Visualization*

## Shell Visualization



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# *Improvements, Successes, and Difficulties*

## What we did well:

- Thorough FEA, CFD, and Topology Optimization
- Unaltered CAD for optimization later
- Communication
- Organization

## Future Improvements:

- Focused on optimizing wide range of parts, did not have time to implement them into CAD
- Phase out OneDrive (not good for CAD structures and file sharing) for Teamcenter, etc.
- More robust scaling factor
- More CFD at different angles
  - Rear wing addition, other technical improvements

## Difficulties

- Version control for CAD parts and assemblies
- Breaking or replacing references when files moved for organizational purposes or scaled
- File sharing system made development slower
- Scaling factor, when changed, would randomly break some constraints in assemblies
- Difficult time constraints prevented implementation of optimized parts into assembly for accurate weights

# *THANK YOU*