

Finite Element Simulation For Mechanical Design



Simple contact analysis

Prof. Andrea Bernasconi Dr. Luca M. Martulli

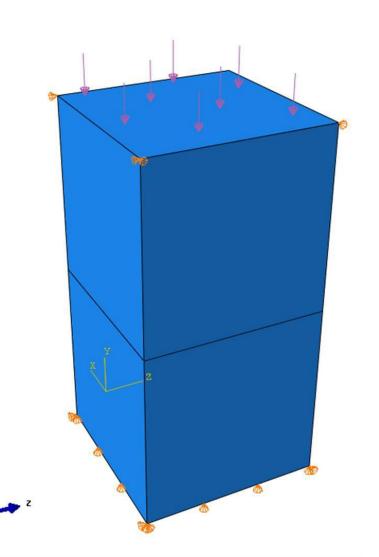
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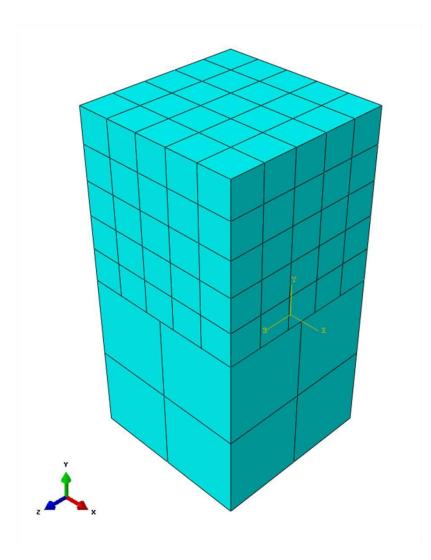
Cube 10x10 mm Pressure 10 MPa

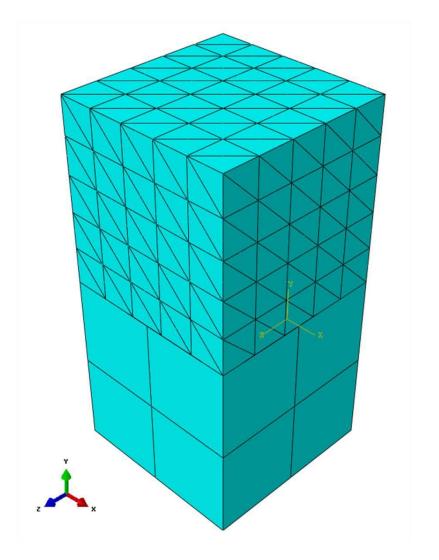
Prevent rigid motions!

cube-on-cube.cae cube-on-cube.jnl

C3D8R_LM_N-to-S_SS
C3D8R_LM_S-to-S_SS
C3D8R_PM_N-to-S_SS
C3D10_PM_N-to-S_SS
C3D10_PM_S-to-S_SS







Contact discretization techniques

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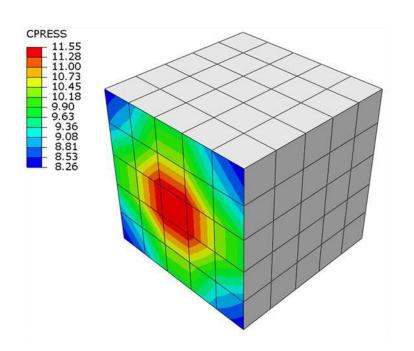
Node-to-Surface

Element type:

C3D8R

Contact Enforcement:

Lagrange Multiplier



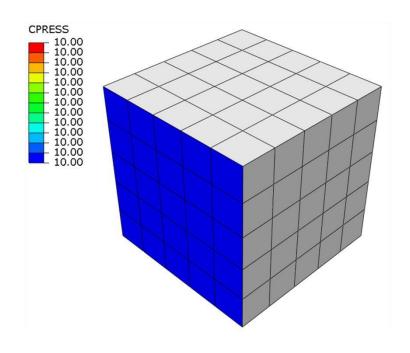
Surface-to-Surface

Element type:

C3D8R

Contact Enforcement:

Lagrange Multiplier



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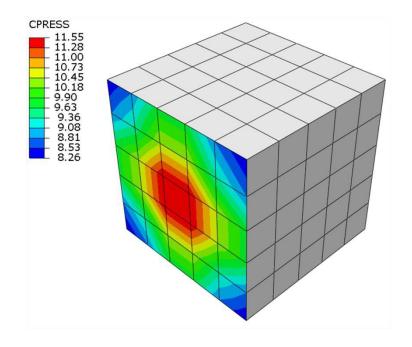
Lagrange Multiplier Method

Element type:

C3D8R

Contact Discretization:

Node-to-Surface



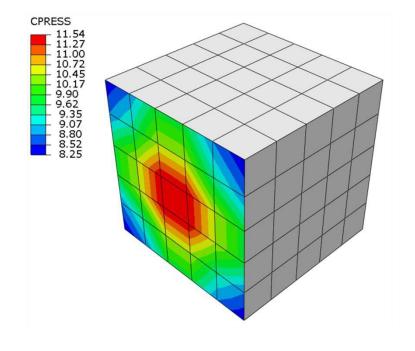
Penalty Method

Element type:

C3D8R

Contact Discretization:

Node-to-Surface



Contact enforcement methods

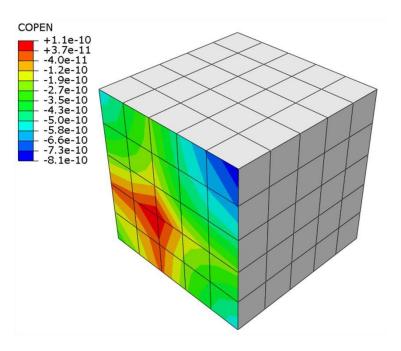
Lagrange Multiplier Method

Element type:

C3D8R

Contact Discretization:

Node-to-Surface



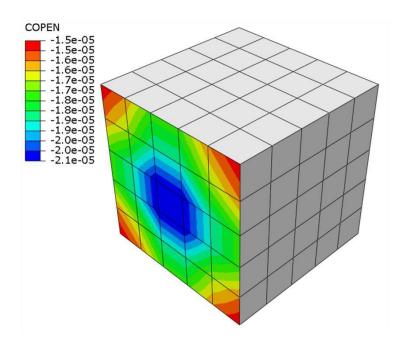
Penalty Method

Element type:

C3D8R

Contact Discretization:

Node-to-Surface







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Contact between two objects

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2D - Disc on block

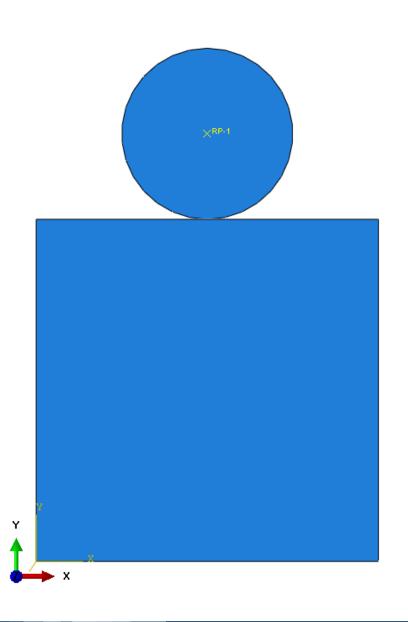
Problem statement

A steel cylinder is pressed into an aluminum block.

Consider a linear elastic behaviour for both materials, and they are homogeneous.

A vertical (y-direction) point load of F=35kN/mm is applied to the cylinder.

A 2D approximation (plane strain) is assumed to be satisfactory.



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2D – Disc on block

Problem statement

Geometry

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Block height = 200 mm

Block width = 200 mm

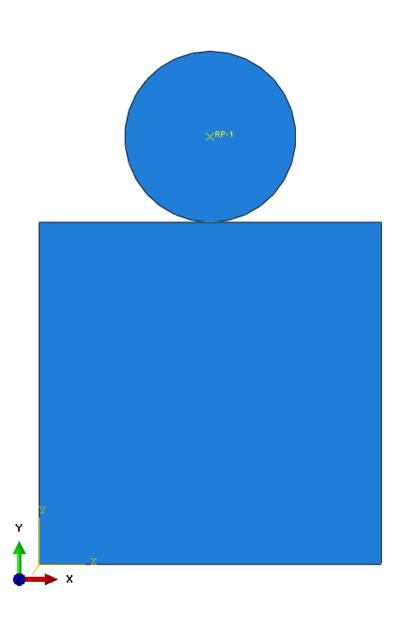
Cylinder diameter = 100 mm

Material properties

 $E_{cyl} = 210 \text{ Gpa}$

E_block = 70 Gpa

 $nu_cyl = nu_block = 0.33$



2D – Disc on block

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Analytical solution

Maximum pressure

$$p_{H} = \sqrt{\frac{W E_{12}'}{\pi \rho_{12}}} = 3585 \text{ N/mm}^{2}$$

Semi-width of contact area

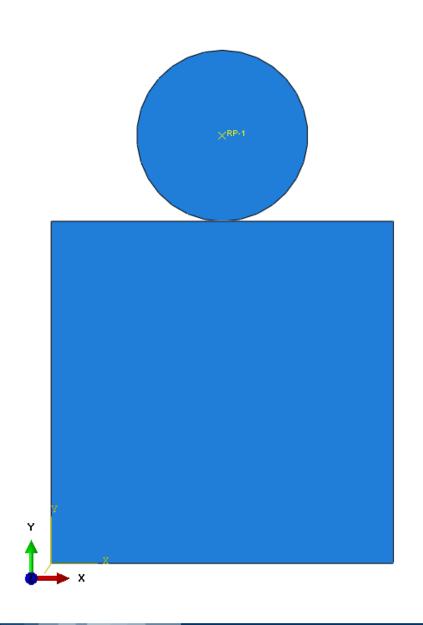
$$b_{H} = 2\sqrt{\frac{W}{\pi} \frac{\rho_{12}}{E'_{12}}} = 6,215 \text{ mm};$$

where

Equivalent radius of curvature

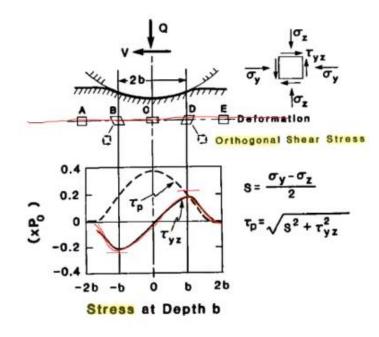
Equivalent elastic modulus

Elastic modulus



2D – Disc on block

Shear stresses without friction



$$\begin{split} \max_{(\overline{x},\overline{y})} \left\{ \overline{\tau}_{_{\text{H}}} \right\} &= 0.300 \text{ at } \overline{x} = 0.00, \overline{y} = 0.78 \\ \max_{(\overline{x},\overline{y})} \left\{ \overline{\tau}_{_{\text{os}}} \right\} &= 0.256 \text{ at } \overline{x} = 0.85, \overline{y} = 0.50 \end{split}$$

Guidelines for choosing master/slave

When a contact pair contains two surfaces, the two surfaces are not allowed to include any of the same nodes and you must choose which surface will be the slave and which will be the master.

For simple contact pairs consisting of two deformable surfaces, the following basic guidelines can be used:

- The larger of the two surfaces should act as the master surface.
- If the surfaces are of comparable size, the surface on the stiffer body should act as the master surface.
- If the surfaces are of comparable size and stiffness, the surface with the coarser mesh should act as the master surface.