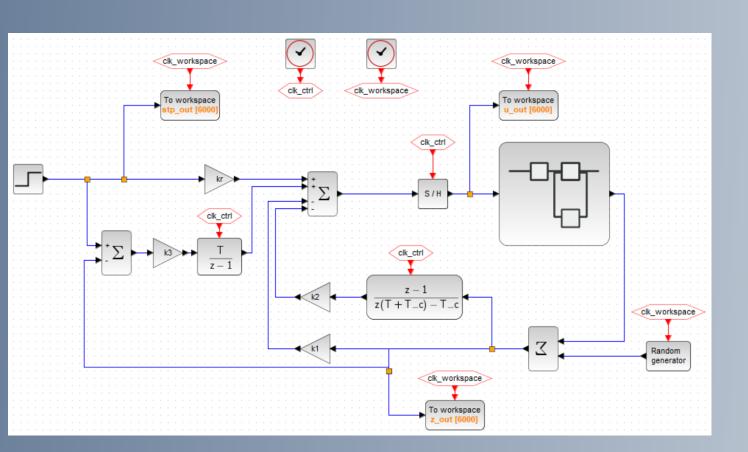
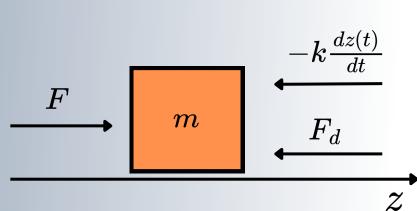
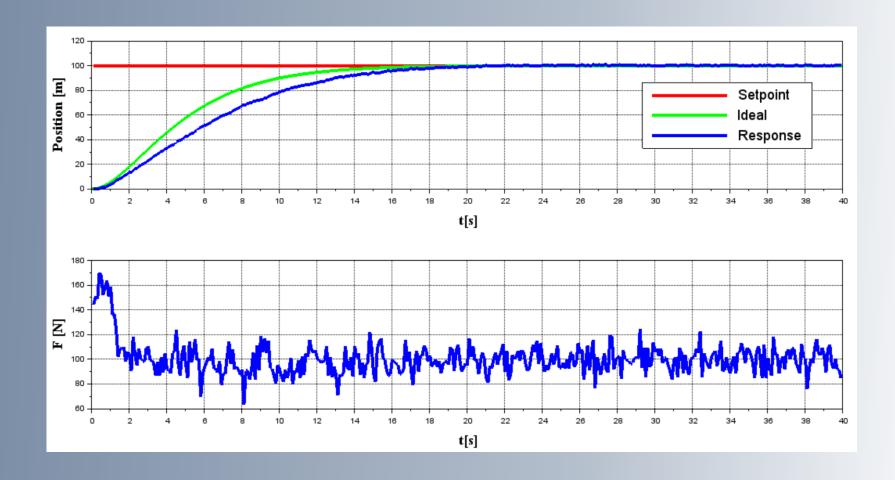
State feedback with integral

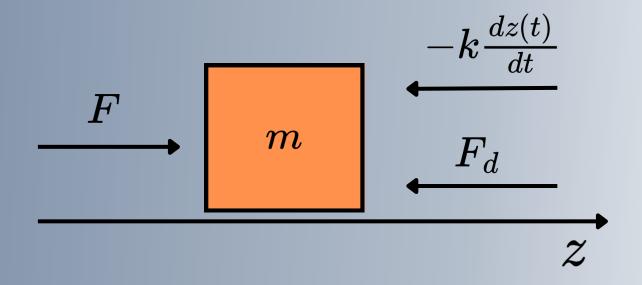








Plant model



$$mrac{d^2z(t)}{dt^2}=F-krac{dz(t)}{dt}-F_d$$

$$m=10kg$$
 $k=0.5rac{Ns}{m}$

Control law - 1

Choose a state-feedback control law that includes the integral of the error:

$$F=k_rr-k_1z-k_2\dot{z}+k_3\int(r(t)-z(t))$$

Substitute in the equation of motion, neglecting the disturbance F_d that is unknown:

$$m\ddot{z}=k_rr-k_1z-k_2\dot{z}+k_3\int(r(t)-z(t))-k\dot{z}$$

Switch to the frequency domain and find the transfer function from r to z:

$$rac{Z(s)}{R(s)} = G_c(s) = rac{srac{k_r}{m} + rac{k_3}{m}}{s^3 + rac{k_2 + k}{m}s^2 + rac{k_1}{m}s + rac{k_3}{m}}$$

Control law - 2

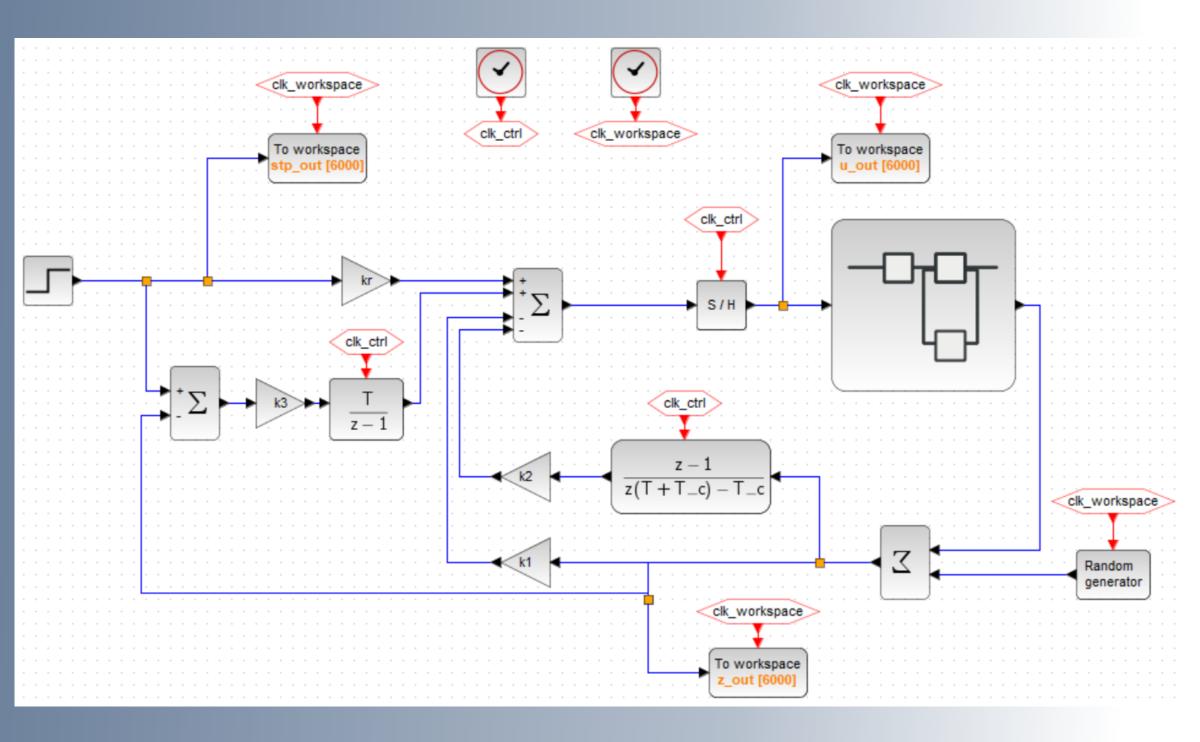
Choose a desired transfer function with the same form, with time to 90% = 10 seconds and steady-state gain = 1:

$$T(s) = rac{0.15s + 0.064}{s^3 + 1.2s^2 + 0.48s + 0.064}$$

Determine k_r , k_1 , k_2 , k_3 so that $G_c(s) = T(s)$:

$$k_r = 0.15m = 1.5$$
 $k_1 = 0.48m = 4.8$
 $k_2 = 1.2m - k = 11.5$
 $k_3 = 0.064m = 0.64$

Control architecture in Xcos



Simulation result

