

Loop-shaping design

A feedback control system configuration

Structure of the cascade controller to be designed

Time domain requirements translation: resume

A three stage procedure

Loop-shaping design

Steady-state controller gain design

Phase lead controller design

Phase lag controller design

Application to dc-motors position control design

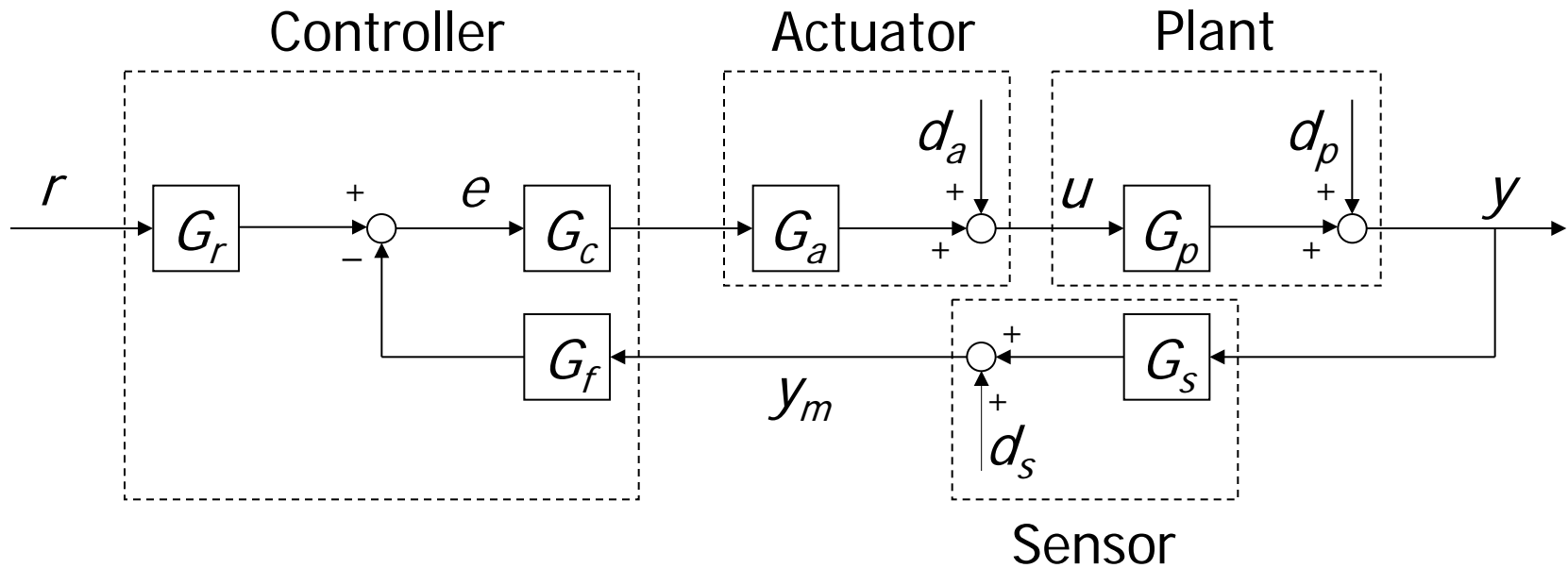
Loop-shaping design

A feedback control system configuration

Output feedback control system configuration

The given parts of the configuration we are dealing with are:

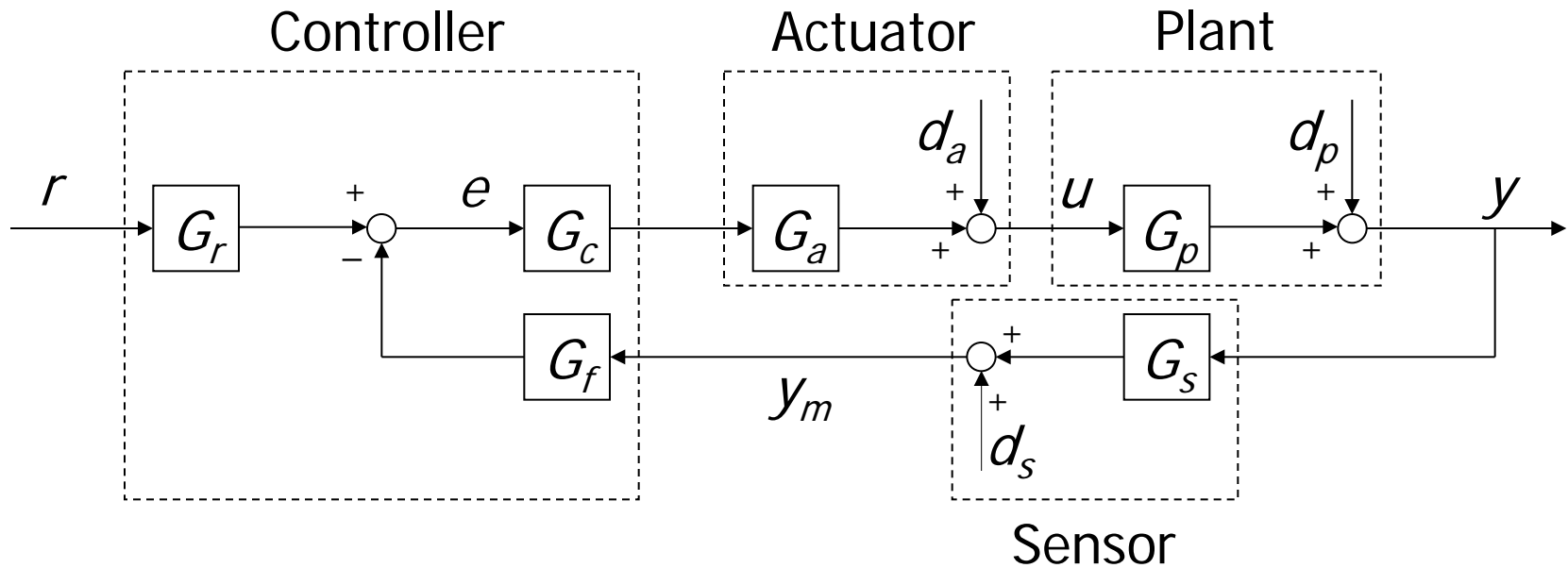
- **plant** G_p with plant disturbance d_p
- **actuator** G_a with actuator disturbance d_a
- **sensor** G_s with sensor noise d_s



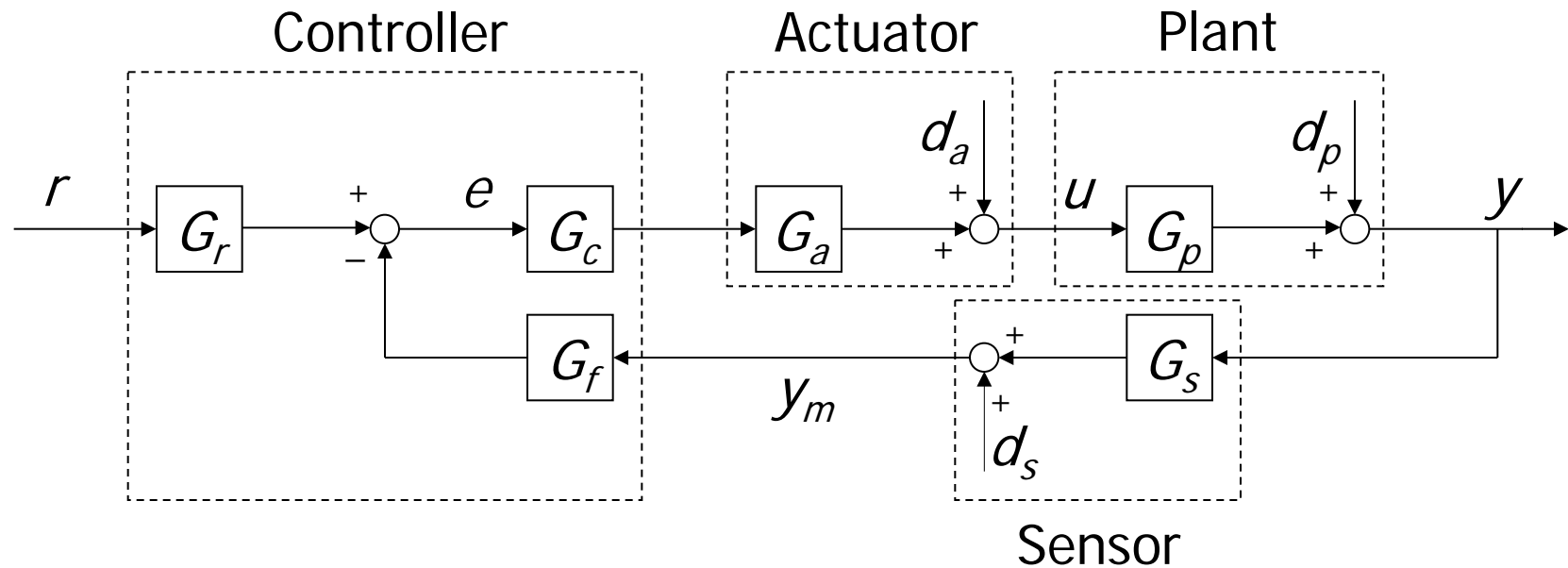
Output feedback control system configuration

The controller to be designed consists of:

- **prefilter** G_r (reference generator)
- **cascade controller** G_c
- **feedback controller** G_f (for 2 DOF or, if constant, for dc gain)



Output feedback control system configuration



Without loss of generality we assume that

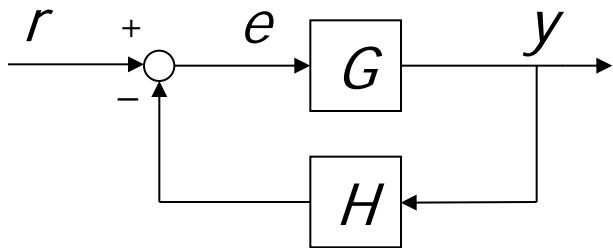
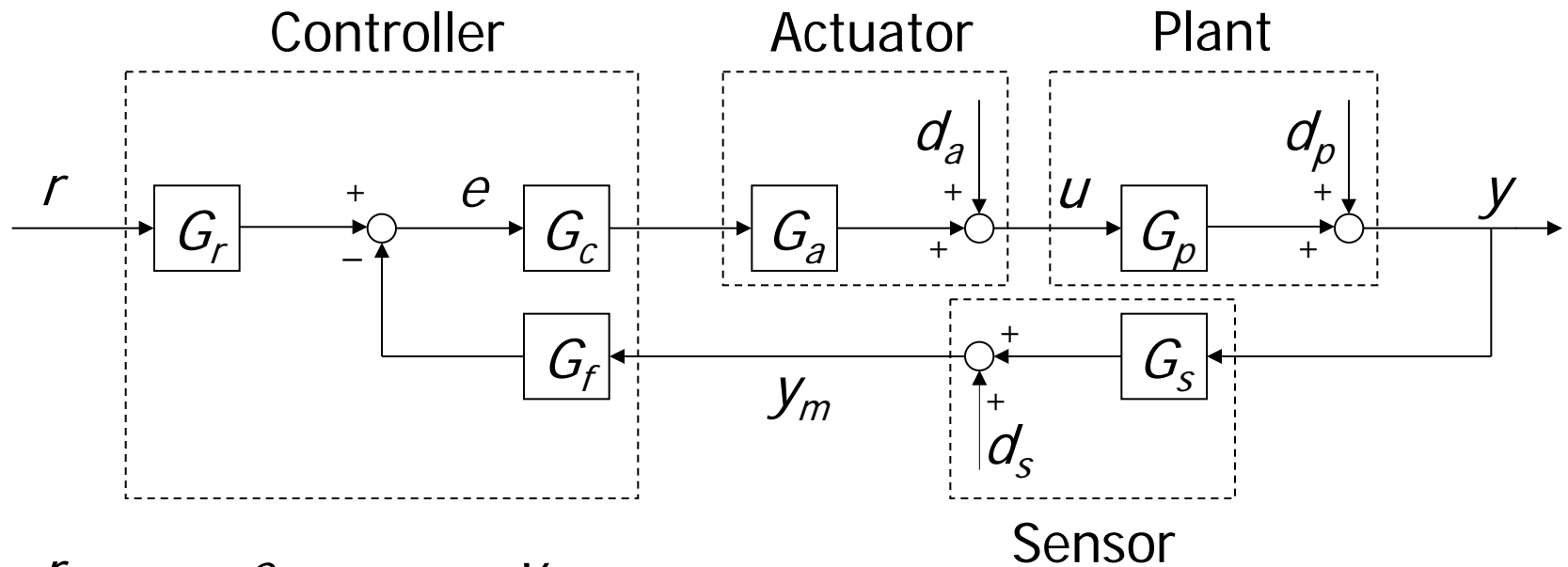
$$G_r = 1$$

G_a = constant (large bandwidth because of fast dynamics)

G_f = constant (to handle the control system steady-state gain)

G_s = constant (large bandwidth because of fast dynamics)

Output feedback control system configuration



$$G = G_c G_a G_p = \frac{N_G}{D_G} \quad H = G_s G_f = \frac{N_H}{D_H}$$

$$\begin{aligned} G_r &= 1 \\ G_a &= \text{constant} \\ G_f &= \text{constant} \\ G_s &= \text{constant} \end{aligned}$$

Loop-shaping design

Structure of the cascade controller to be designed

- LTI controllers described by the transfer function

$$G_c(s) = \frac{K_c}{s^\nu} \prod_i \left(\frac{1 + \frac{s}{z_{di}}}{1 + \frac{s}{m_{di} z_{di}}} \right) \prod_j \left(\frac{1 + \frac{s}{m_{ij} p_{ij}}}{1 + \frac{s}{p_{ij}}} \right)$$

will be considered from now on.

- The controller has ν poles at $s = 0$ and its generalized steady-state gain is defined as:

$$\lim_{s \rightarrow 0} s^\nu G_c(s) = K_c$$

- The plant has p poles at $s = 0$ and its generalized steady-state gain is defined as:

$$\lim_{s \rightarrow 0} s^p G_p(s) = K_p$$

The part $\frac{K_c}{s^\nu}$ is designed from the steady state requirements

Next we will see how the remaining factors can be designed

$$\prod_i \left(\frac{1 + \frac{s}{z_{di}}}{1 + \frac{s}{m_{di} z_{di}}} \right) \prod_j \left(\frac{1 + \frac{s}{m_j p_j}}{1 + \frac{s}{p_j}} \right)$$

from the transient time domain requirements

Loop-shaping design

Time domain requirements translation: resume

Time domain requirements translation: resume

- The 2nd order prototype model

$$T(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

is used to translate the transient time domain requirements

$$\hat{s} \quad t_r \quad t_{s,\alpha\%}$$

into the relevant indices of the frequency response of the functions $T(s)$, $S(s)$ and $L(s)$...

Time domain requirements translation: resume

... in particular:

$$\hat{S} \rightarrow \begin{cases} T_p \rightarrow \text{resonance peak of } |T(j\omega)| \\ S_p \rightarrow \text{resonance peak of } |S(j\omega)| \end{cases}$$

$$\begin{matrix} t_r \\ t_{s,\alpha\%} \end{matrix} \rightarrow \omega_c \rightarrow \text{crossover frequency of } |L(j\omega)|$$

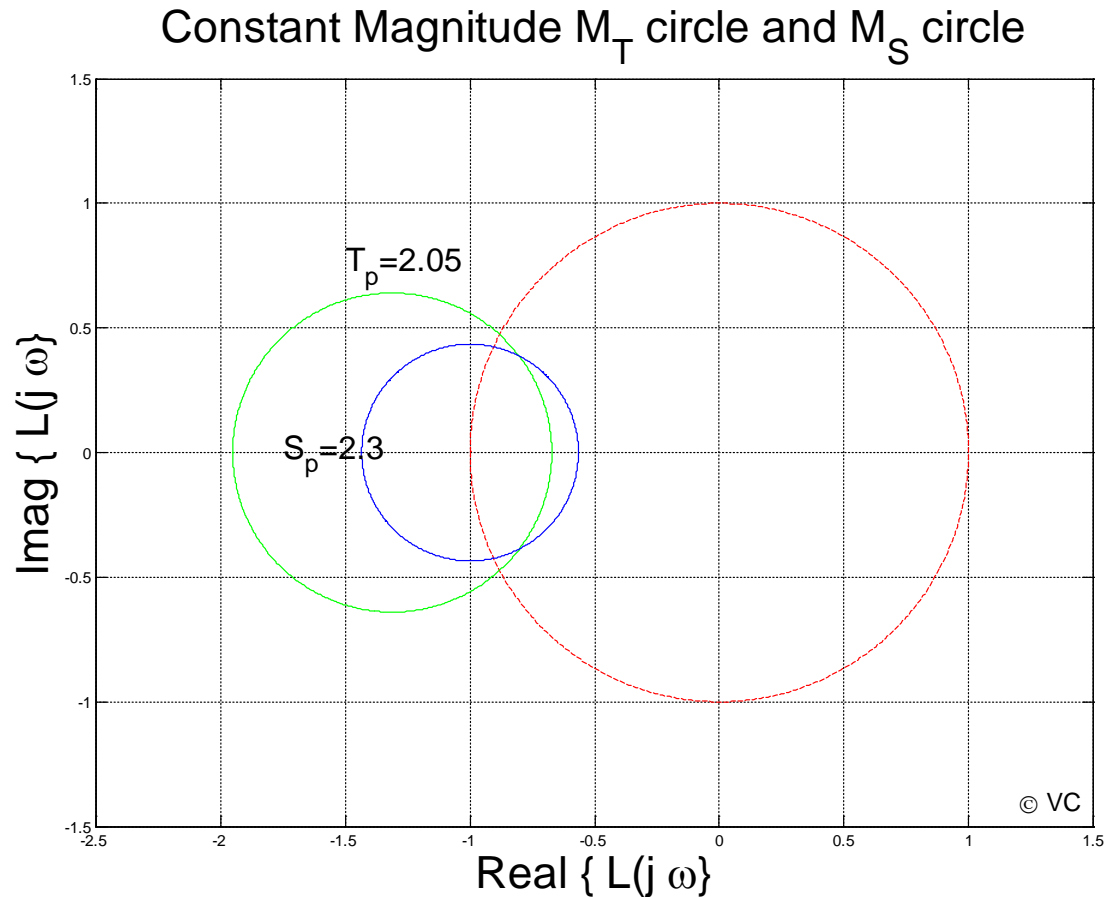
Time domain requirements translation: resume

It is easy to show that S_p and T_p are related to the gain and phase margins.

$$\text{GM} \geq \frac{S_p}{S_p - 1}, \quad \text{PM} \geq 2 \sin^{-1} \left(\frac{1}{2S_p} \right)$$

$$\text{GM} \geq 1 + \frac{1}{T_p}, \quad \text{PM} \geq 2 \sin^{-1} \left(\frac{1}{2T_p} \right)$$

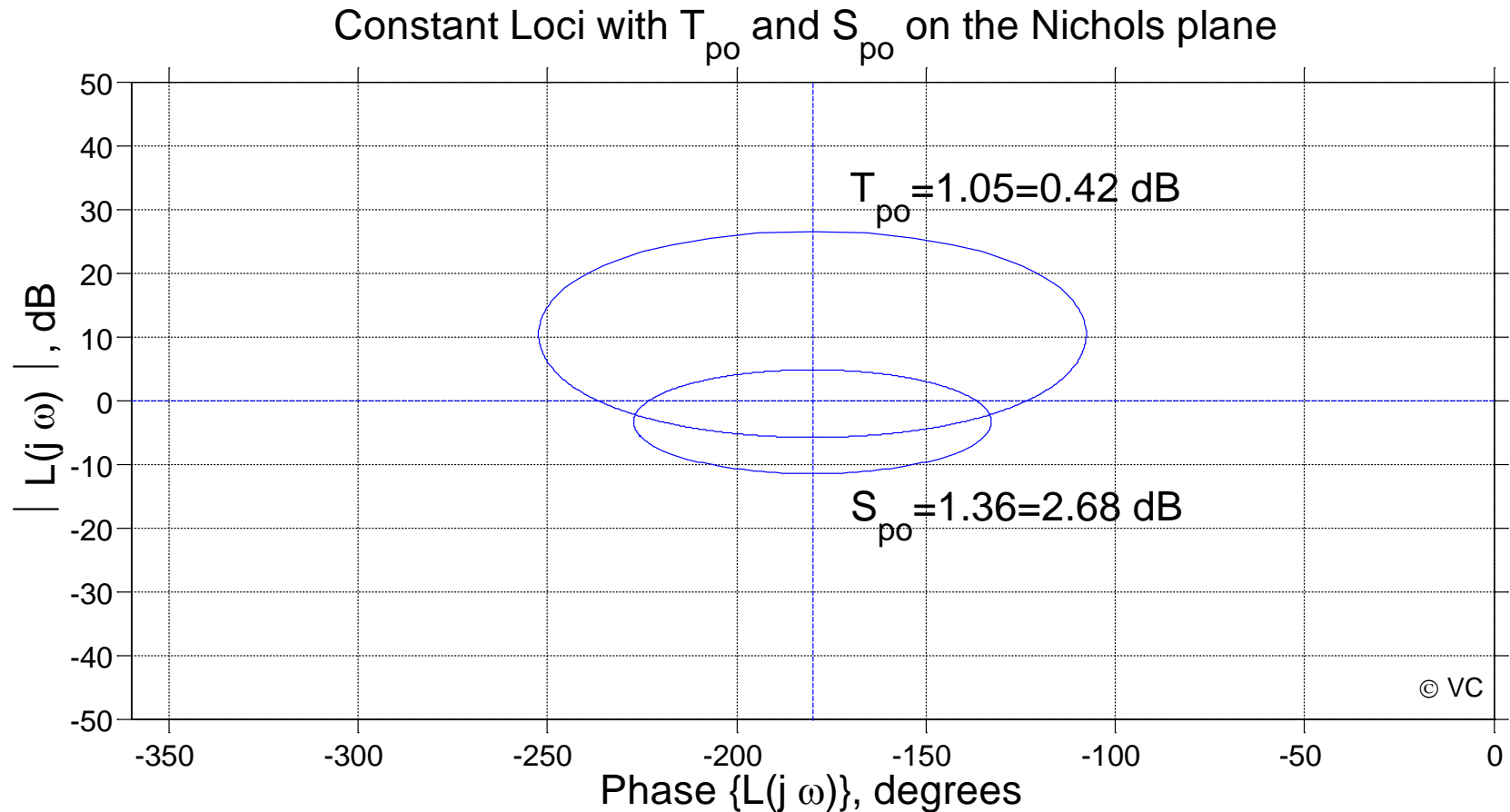
Time domain requirements translation: resume



Time domain requirements translation: resume

- The values of the resonance peaks T_p and S_p of $|T(j\omega)|$ and $|S(j\omega)|$ respectively obtained via the \hat{S} requirement can be used to draw on the Nichols plane the corresponding constant magnitude loci
- Such loci can be considered as constraints which should not be violated by the Nichols plot of the frequency response of the loop function $L(j\omega)$

Time domain requirements translation: resume



Loop-shaping design

A three stage procedure controller design

A three stage procedure controller design

The controller design is accomplished through the following three stage procedure:

Stage 1: Steady-state and transient requirements translation

Stage 2: Controller design through loop-shaping techniques

Stage 3: Performance analysis of the designed feedback control system, focusing the report on the assigned requirements. If some of the requirements are not fulfilled, go back to stage 2 and modify the designed controller accordingly.

Loop-shaping design

Steady-state controller gain design

Steady-state controller gain design

First, according to the control system type derived from the steady-state requirements, a suitable number of poles at the origin (v) must be set in the controller.

Thanks to the stability Nyquist criterion, the correct sign of K_c is chosen.

Then, the Nichols plot of the loop function frequency response should be drawn on the phase-magnitude plane (the Nichols plane).

An increase of $\text{abs}(K_c)$ shifts the Nichols plot upwards.

A decrease of $\text{abs}(K_c)$ shifts the Nichols plot downwards.

Steady-state controller gain design

The $\text{abs}(K_c)$ should be greater than the minimum value derived from the steady-state requirements.

The $\text{abs}(K_c)$ should be less than the value which makes the Nichols plot of the loop function frequency response intersect the constant loci corresponding to the resonance peaks T_p and S_p .

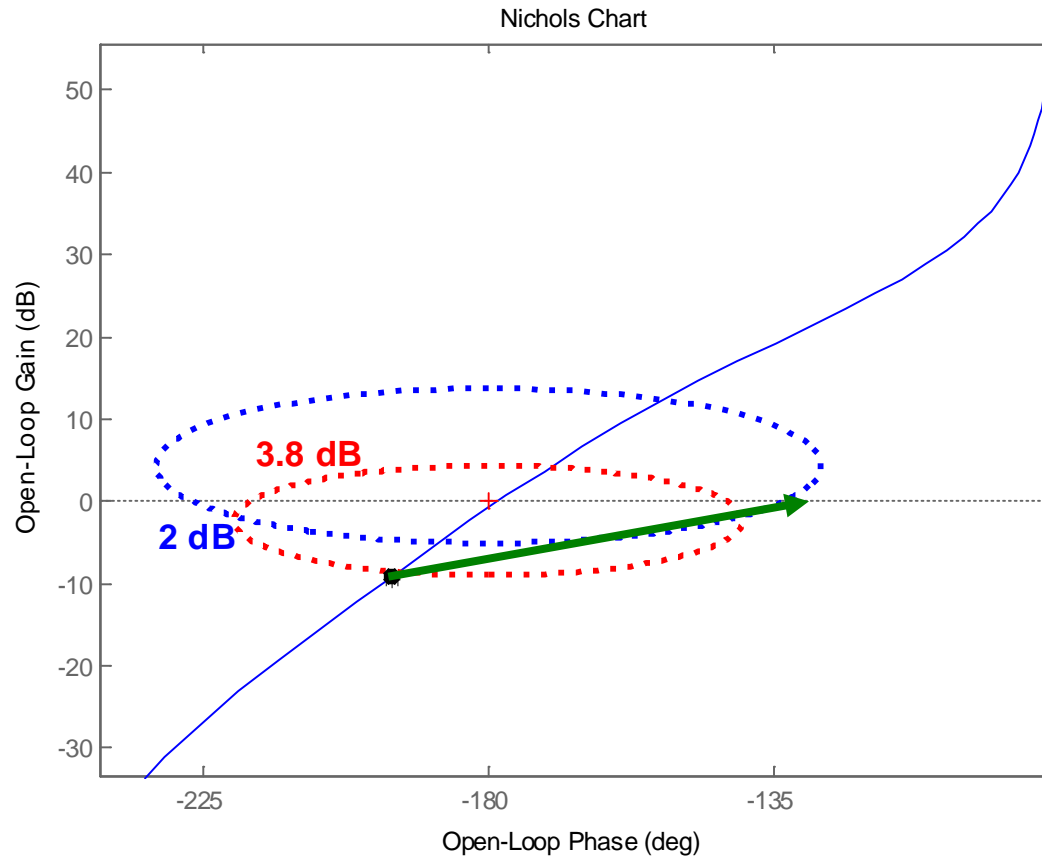
When we move the $\text{abs}(K_c)$, the shape of the loop function frequency response is not modified.

Higher values of $\text{abs}(K_c)$ imply higher crossover frequency (faster response), decreased steady state errors to polynomial references and disturbances, decreased stability margin.

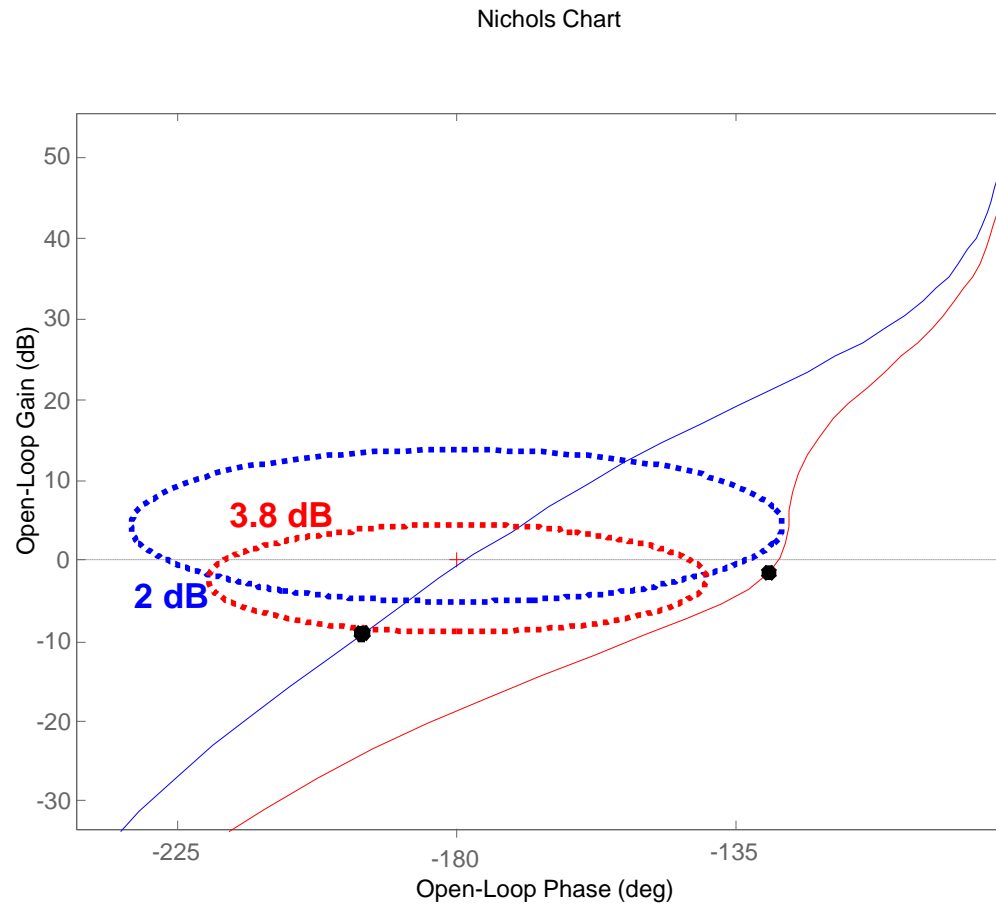
Lower values of $\text{abs}(K_c)$ imply lower crossover frequency (slow response), increased steady state errors to polynomial references and disturbances, increased stability margin.

Loop-shaping design

Phase lead controller design



- In this case phase lead and magnitude increase actions are required

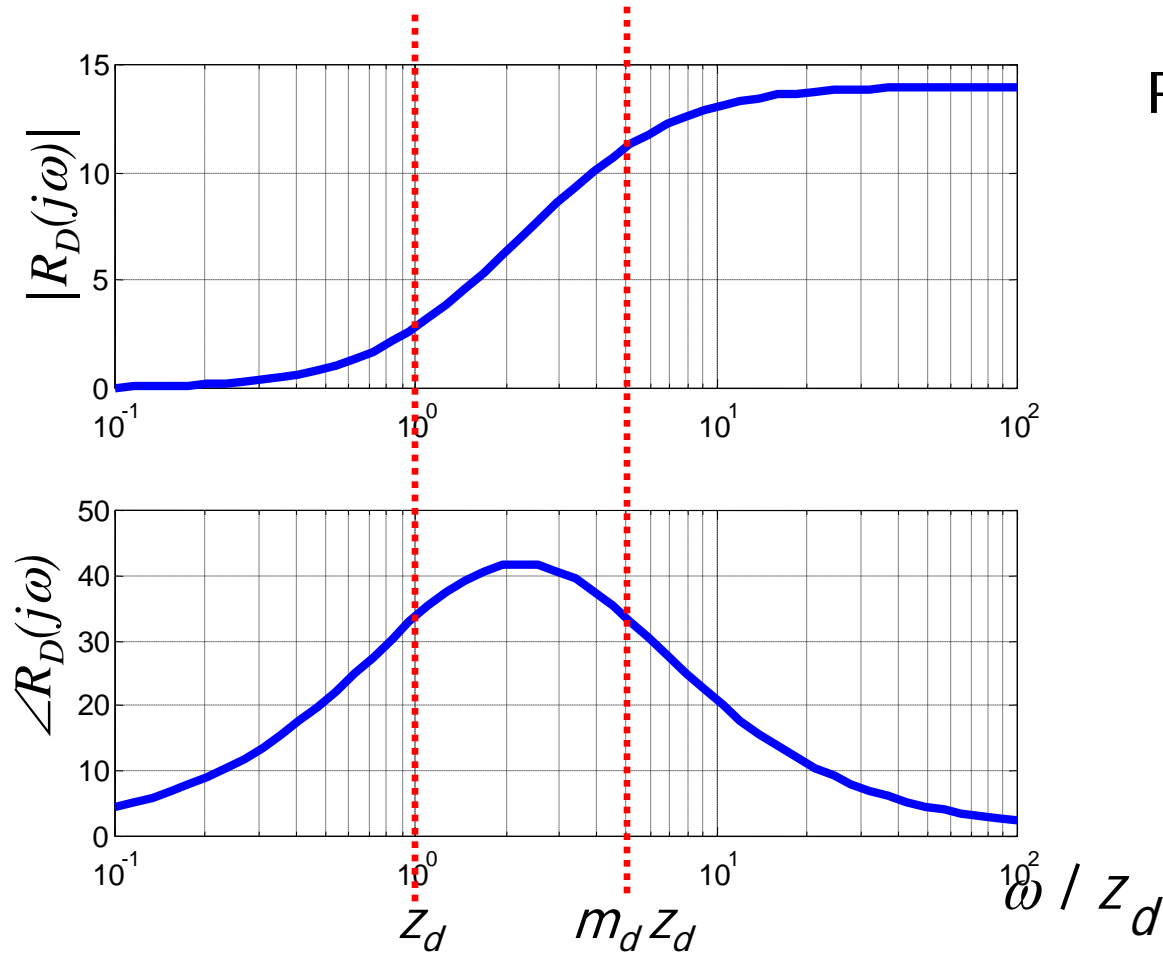


The introduced example motivates the use of the

lead network

$$R_d(s) = \frac{1 + \frac{s}{z_d}}{1 + \frac{s}{m_d z_d}}, m_d > 1$$

The lead network: frequency response

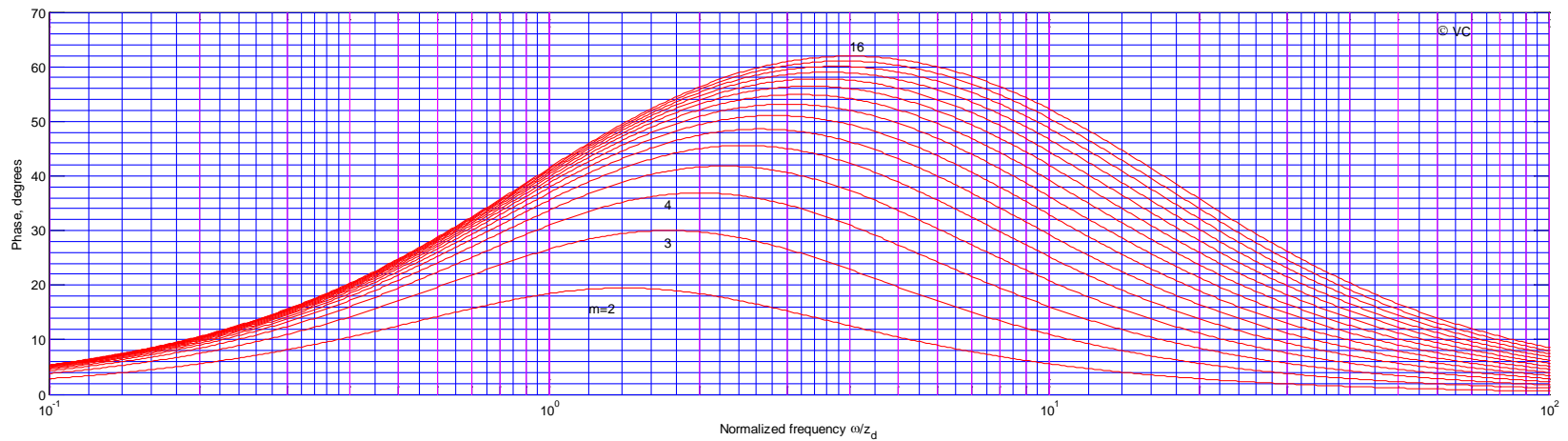
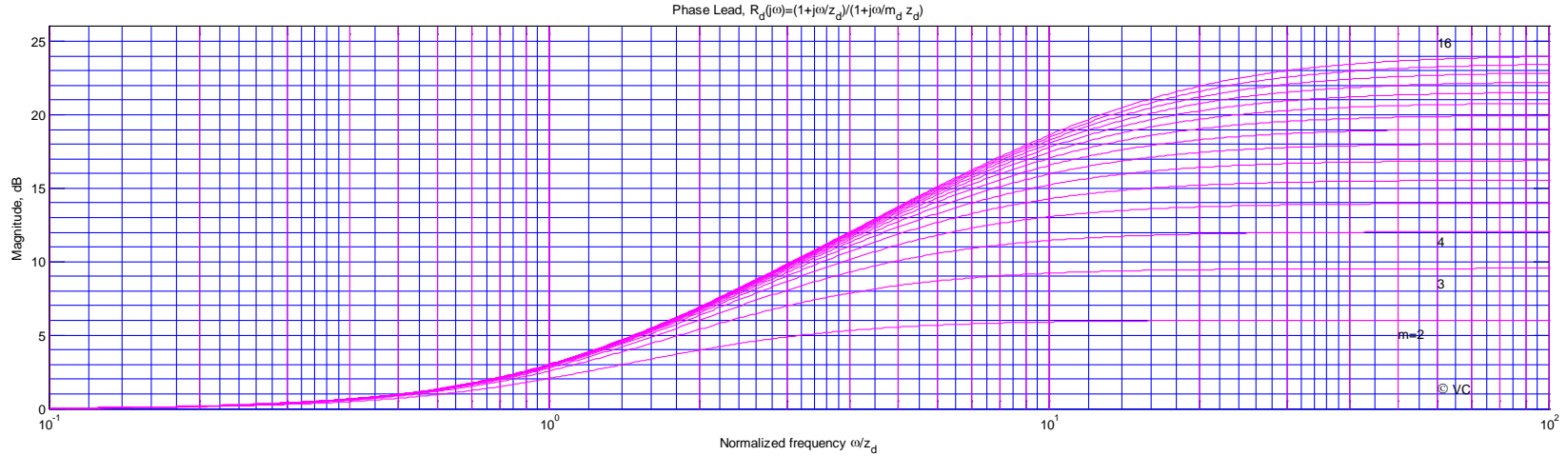


$$R_d(s) = \frac{1 + \frac{s}{z_d}}{1 + \frac{s}{m_d z_d}}, m_d > 1$$

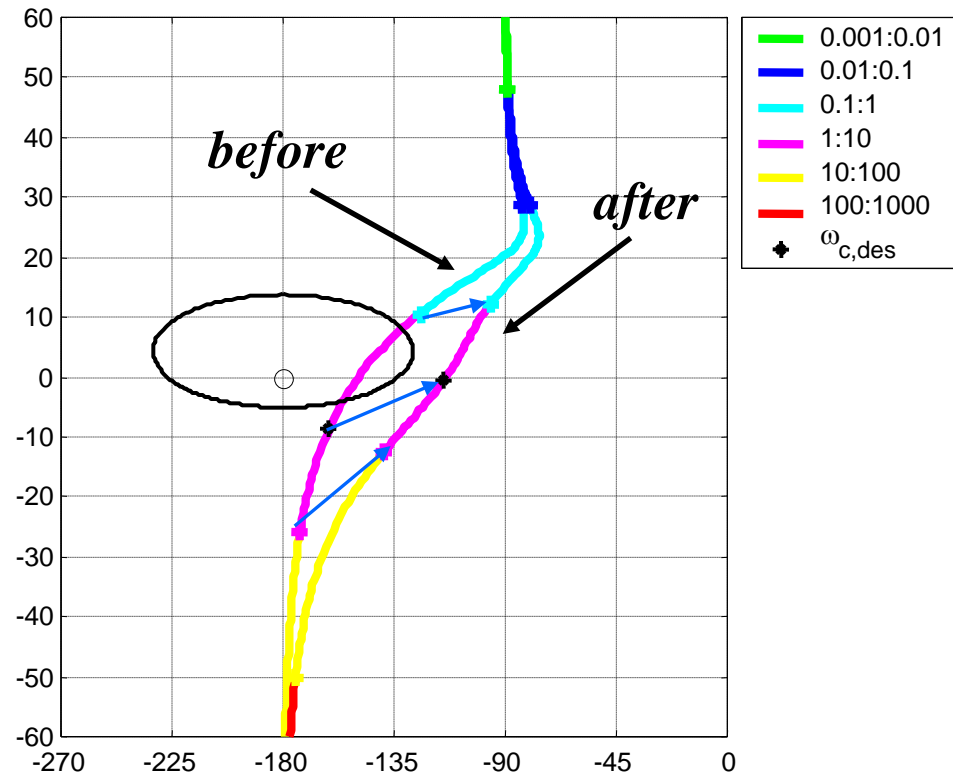
Magnitude increase

Phase lead

Phase lead frequency response



The lead network: effects



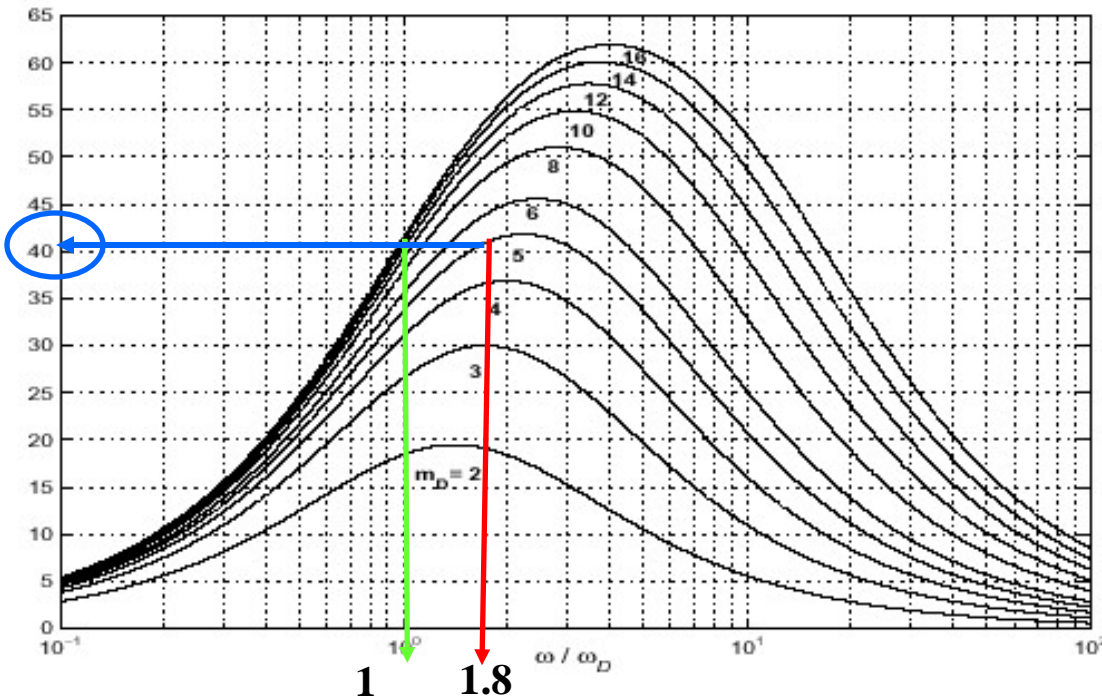
Effects on the Nichols plane:

- phase lead and magnitude increase gives rise to an oblique shift of the loop Nichols plot over the frequencies of interest.

The lead network: design

- Suppose that a 40° phase lead is required at $\omega_{c,des} = 7 \text{ rad/s}$

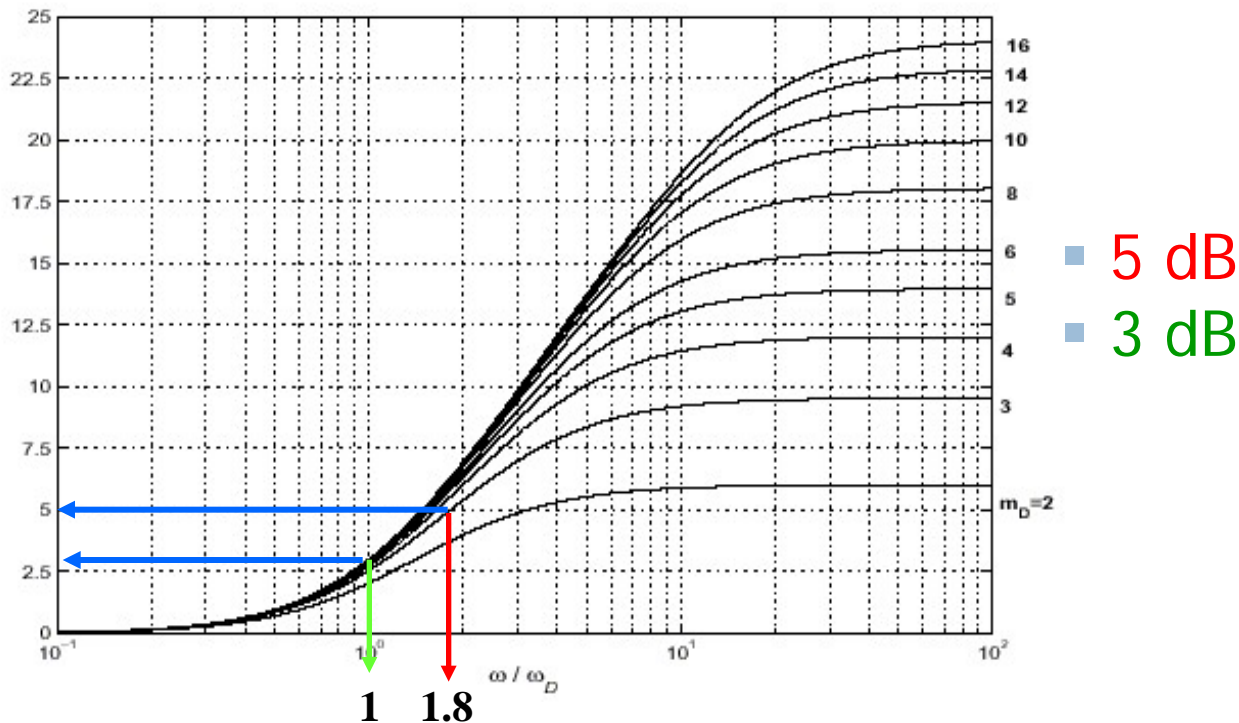
several choices can be made:



- $m_D = 5; \omega/z_D = 1.8$
 $\rightarrow (\omega/z_D)|_{\omega=\omega_c} = 1.8$
 $z_D = \omega_c / 1.8 = 3.89 \text{ rad/s}$

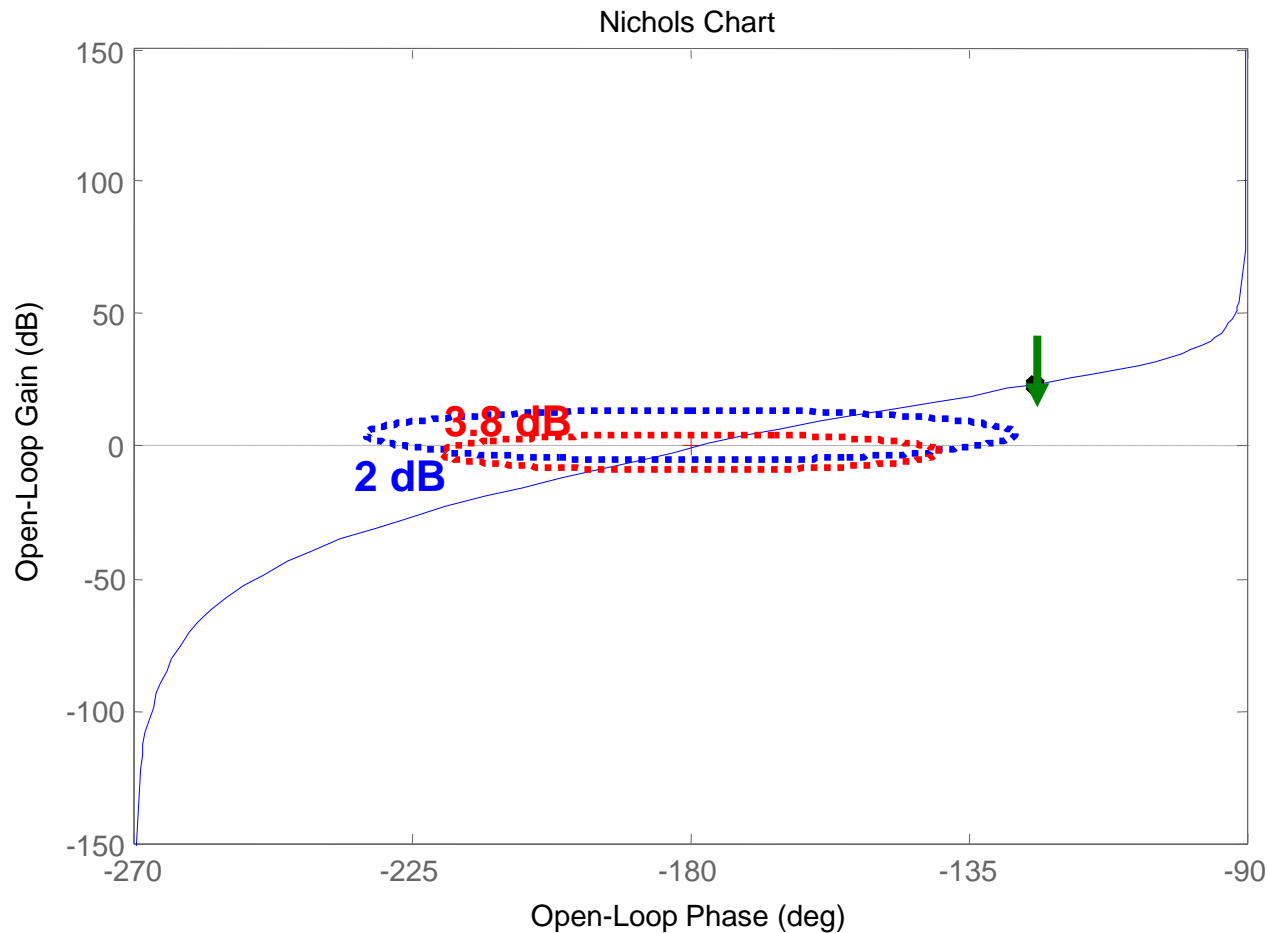
- $m_D = 10; \omega/z_D = 1$
 $\rightarrow (\omega/z_D)|_{\omega=\omega_c} = 1$
 $z_D = \omega_c / 1 = 7 \text{ rad/s}$

- The corresponding magnitude increases are:

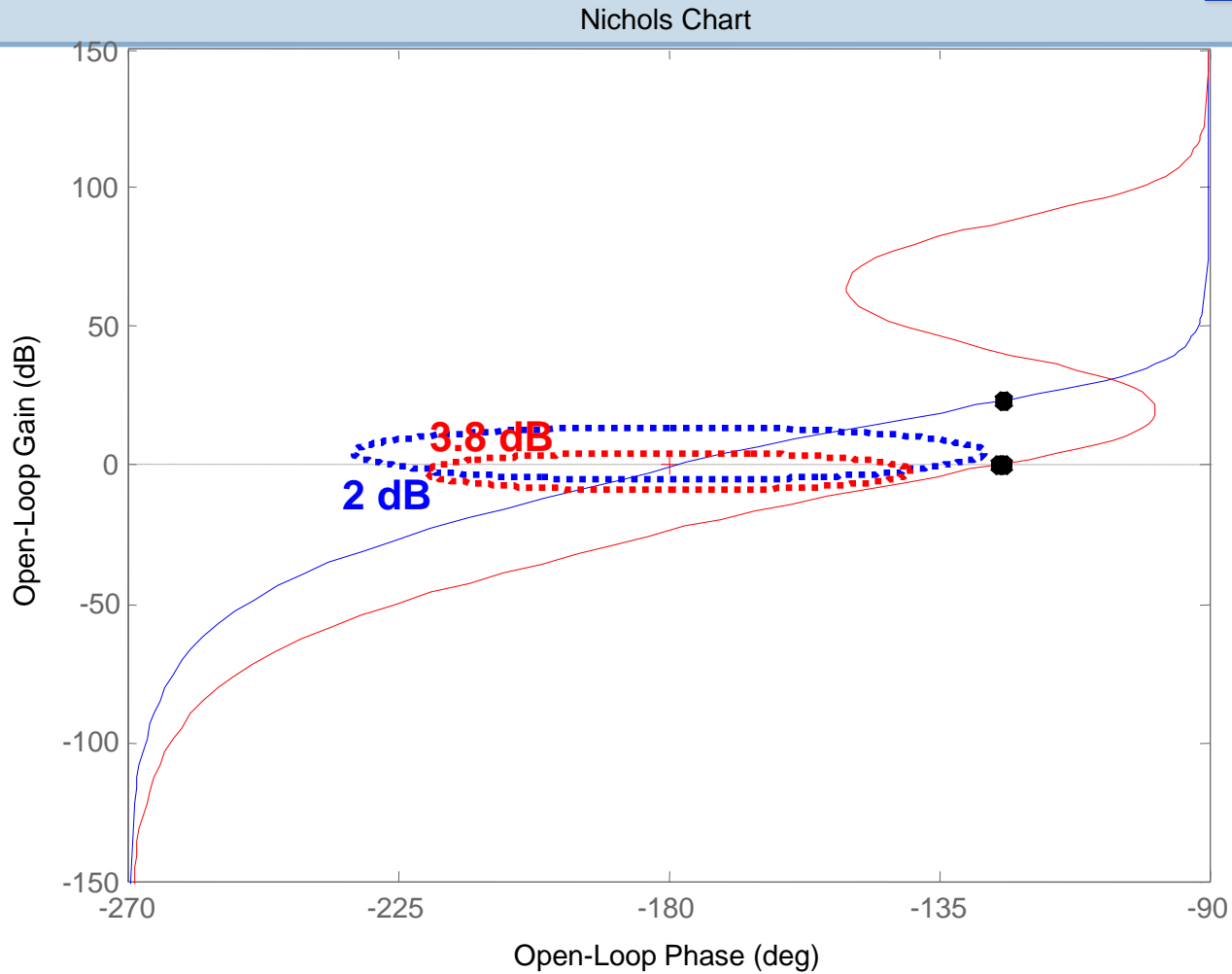


Loop-shaping design

Phase lag controller design



A magnitude attenuation action is required

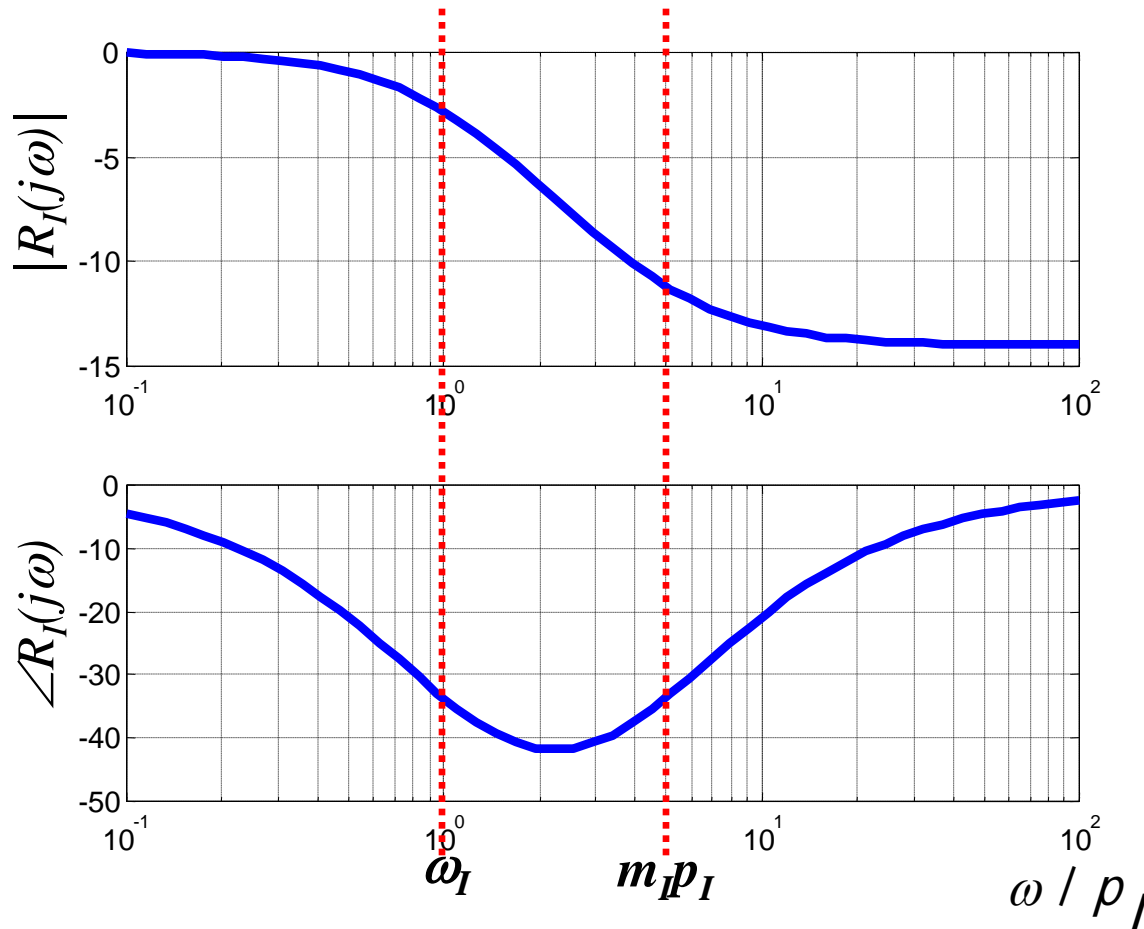


The introduced example motivates the use of the

lag network

$$R_i(s) = \frac{1 + \frac{s}{m_i p_i}}{1 + \frac{s}{p_i}}, m_i > 1$$

The lag network: frequency response

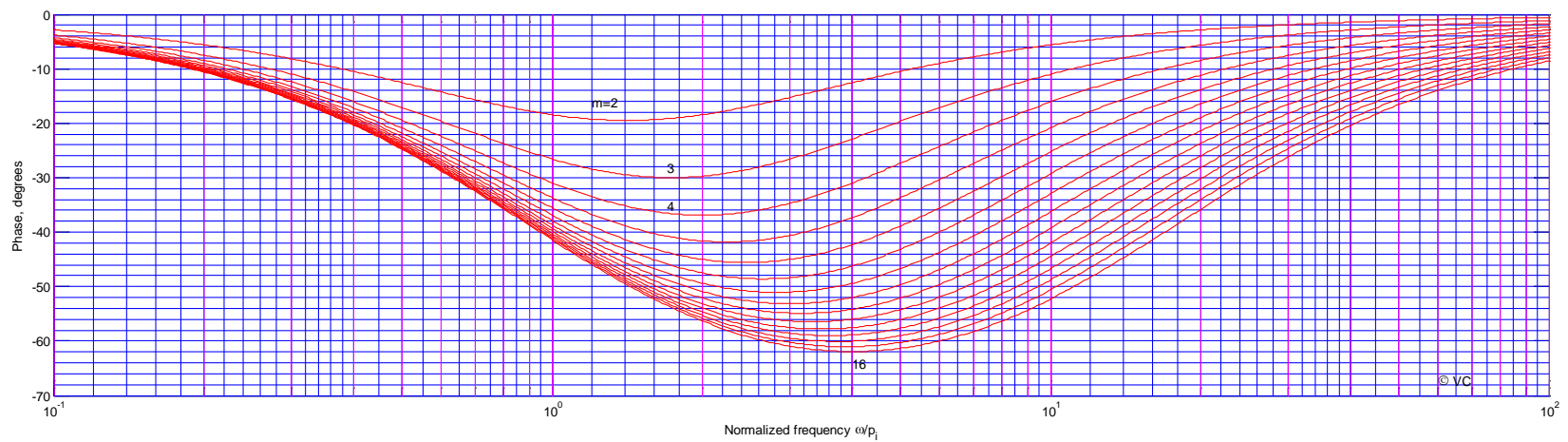
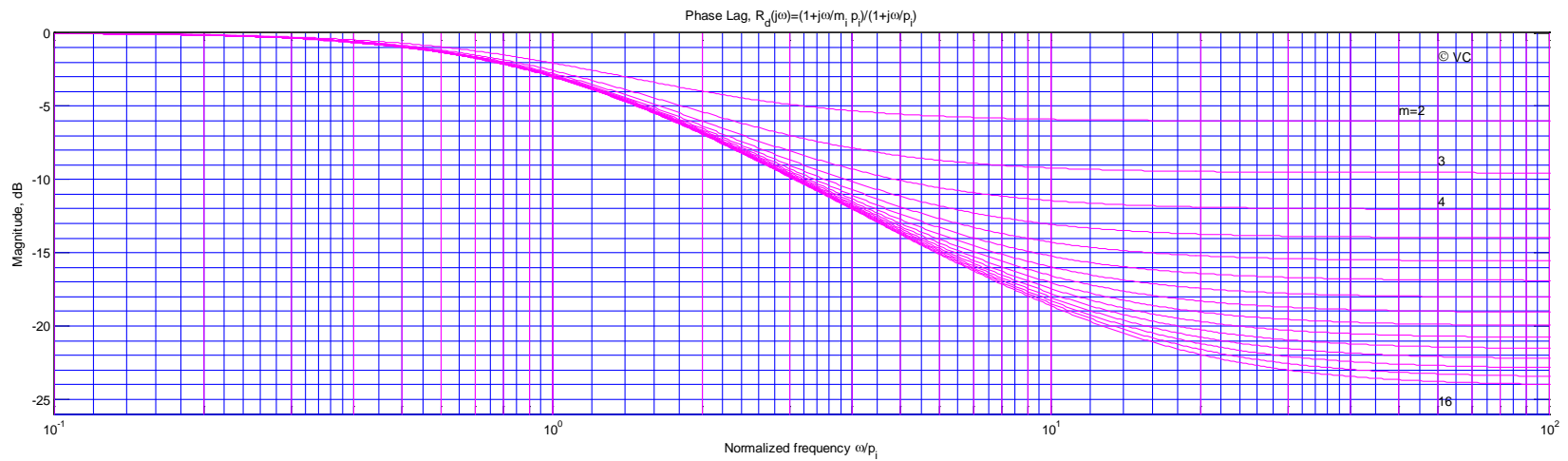


$$R_i(s) = \frac{1 + \frac{s}{m_i p_i}}{1 + \frac{s}{p_i}}, m_i > 1$$

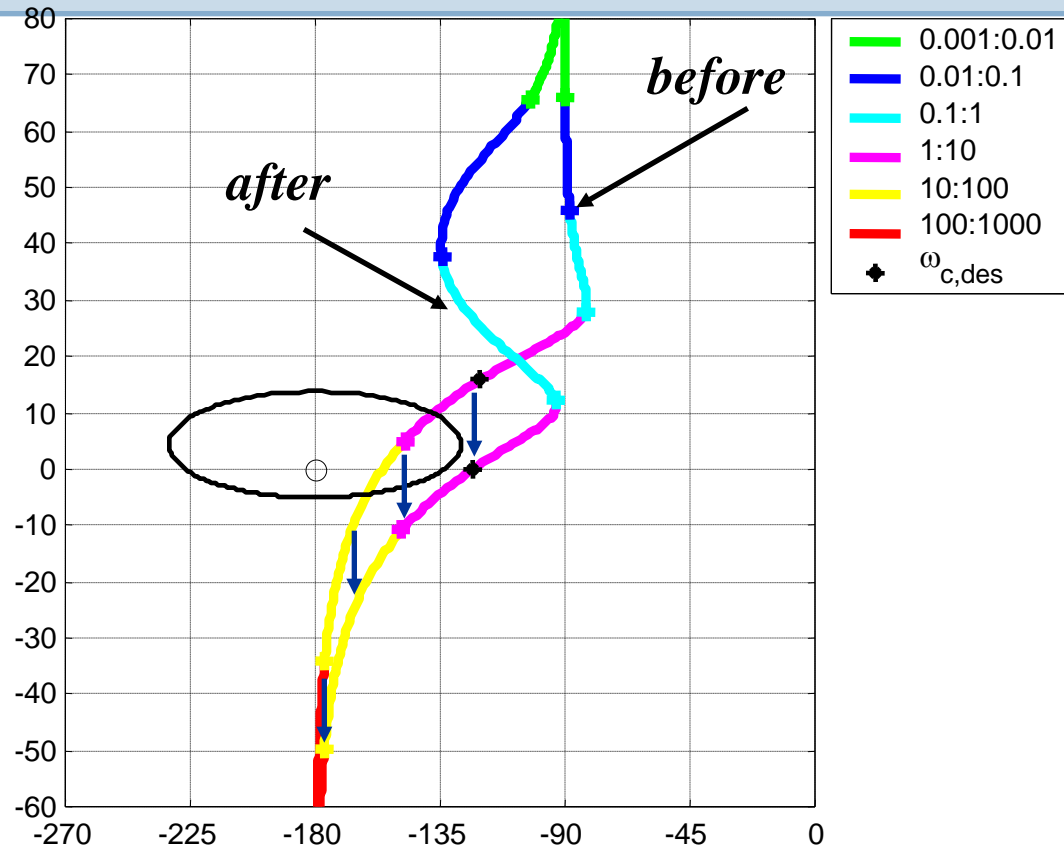
- Magnitude attenuation

- Phase lag

Phase lag frequency response



The lag network: effects

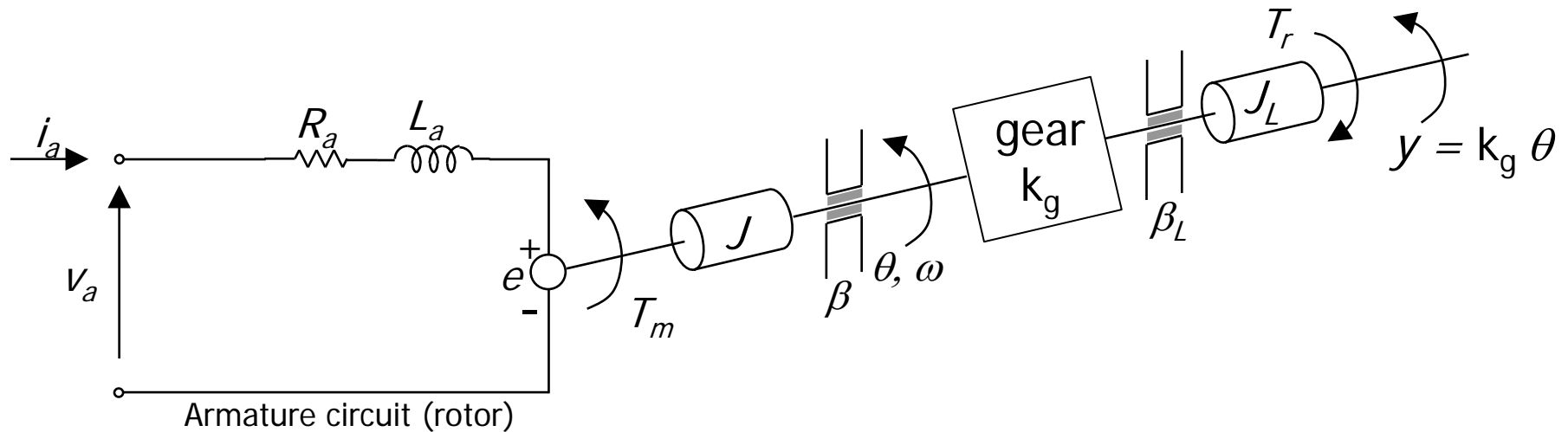


Effects on the Nichols plane:

- the magnitude attenuation effect gives rise to a vertical shift of the loop Nichols plot over the frequency of interest.

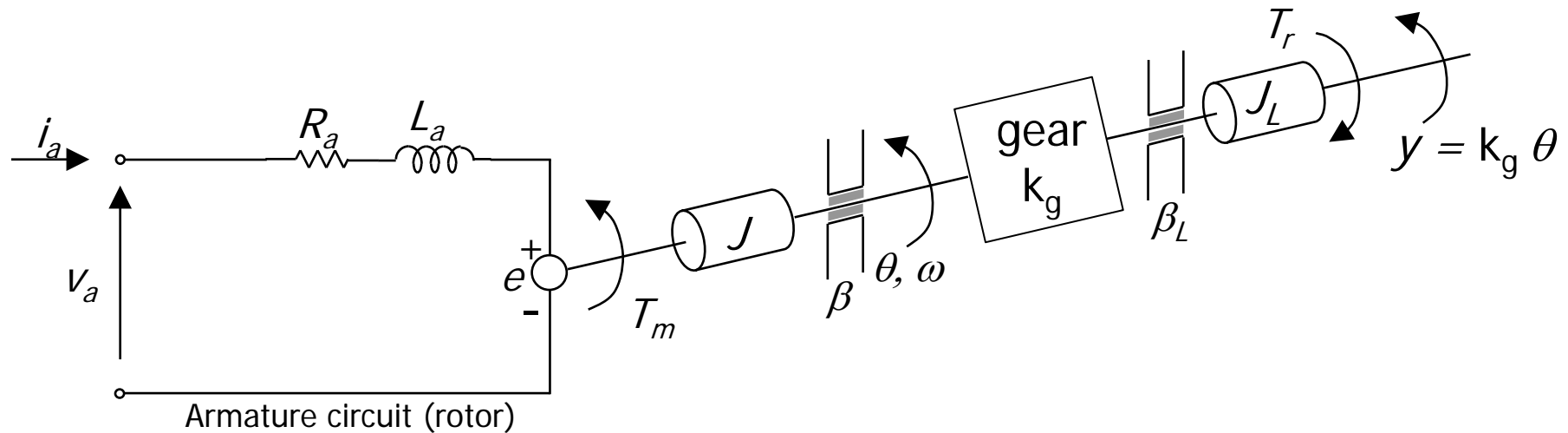
Loop-shaping design

Application to dc-motors position control design



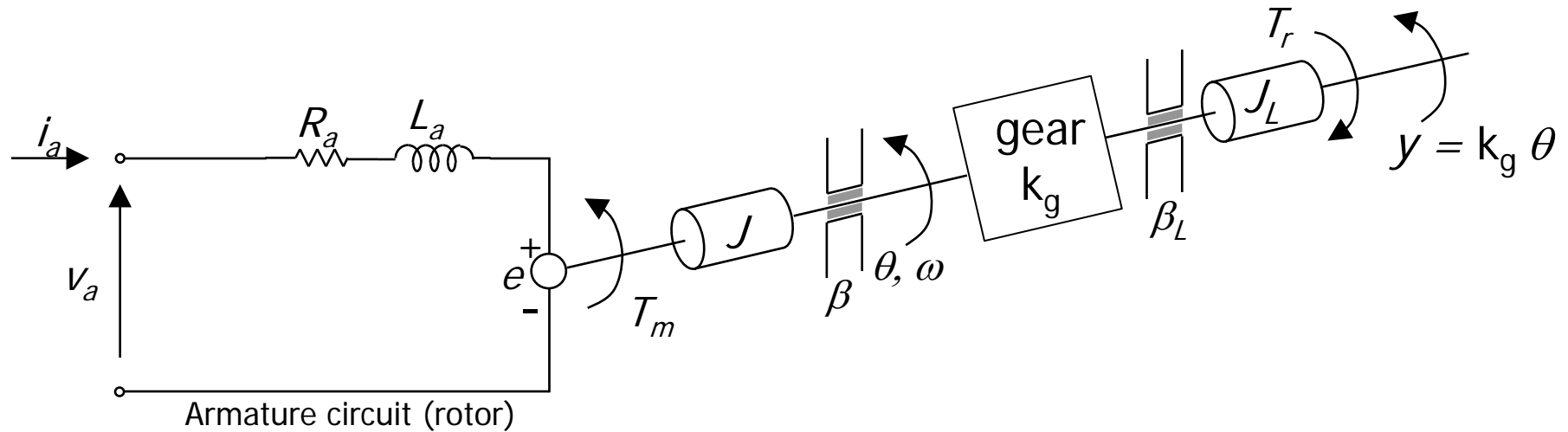
Suppose we are given the plant transfer function:

$$G_p(s) = \frac{Y(s)}{V_a(s)} = \frac{1.61}{s \left(1 + \frac{s}{54.67} \right) \left(1 + \frac{s}{5.49} \right)}$$



... and the disturbance transfer function

$$G_d(s) = \frac{Y(s)}{T_r(s)} = \frac{3.23 \left(1 + \frac{s}{60} \right)}{s \left(1 + \frac{s}{54.67} \right) \left(1 + \frac{s}{5.49} \right)}$$

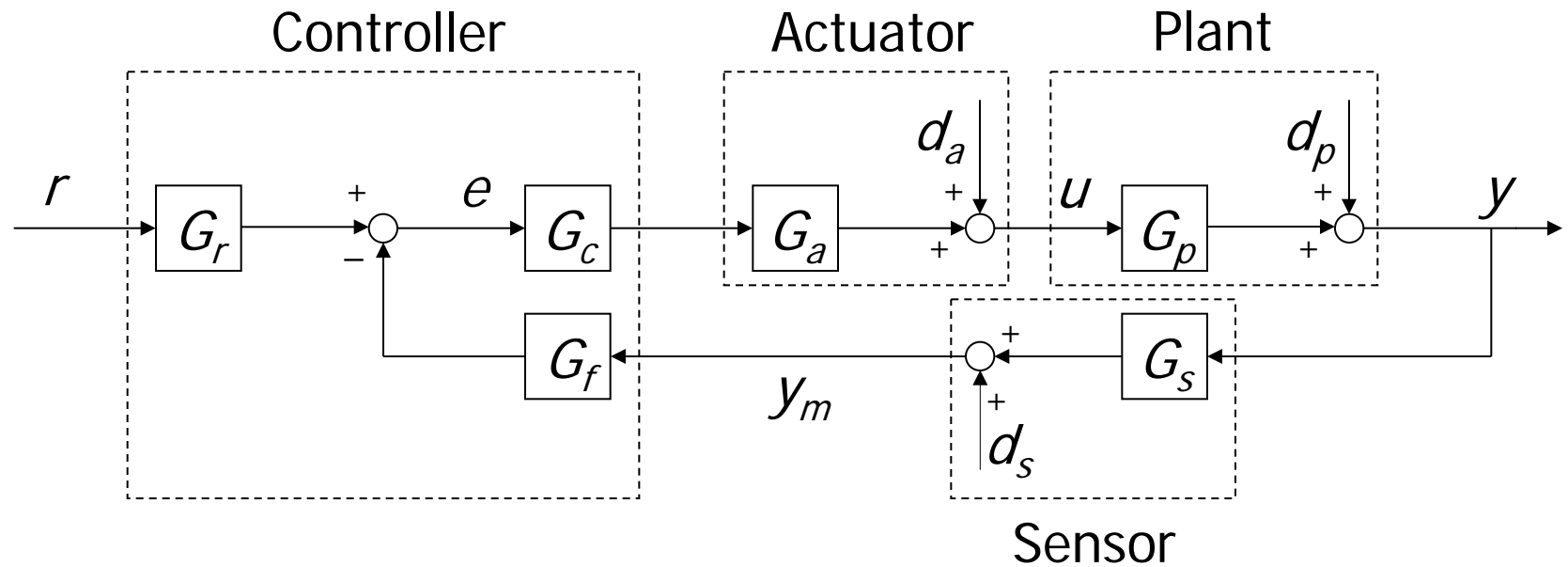


Design a control system such that:

$$\hat{s} \leq 10\%$$

$$|e_r^\infty| \leq \rho_r = 2 \cdot 10^{-2} \quad \text{when} \quad r(t) = R_0 t; \quad |R_0| \leq \frac{\pi}{10}$$

Feedback control systems to be designed



$$\begin{aligned} G_r &= 1 \\ G_a &= 1 \\ K_d &= 1 \\ G_s &= 1 \end{aligned}$$