AEEM4058 - Homework 5

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10.17.2023

Problem 1

Figure 1 shows a beam of 2 m length that is supported by a truss member at the middle point A and clamped on the rigid walls at the two ends. It carries a uniformly distributed vertical load over the left span. The truss and beam are all of uniform cross-section and made of the same material with the Young's modulus E = 200.0 GPa. The cross section of the beam is circular, and the area of the cross-section is 0.01 m^2 . The area of the cross-section of the truss member is 0.0002 m². Using the finite element method with two elements for the entire beam, using an efficient approach to

- (a) calculate the nodal displacement at the middle pint A,
- (b) calculate the reaction forces at the supports for the beam and truss members,
- (c) calculate the internal forces in the truss member.

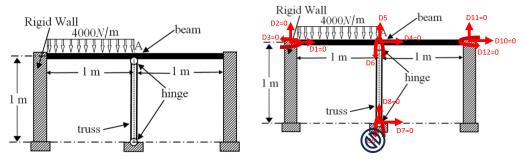
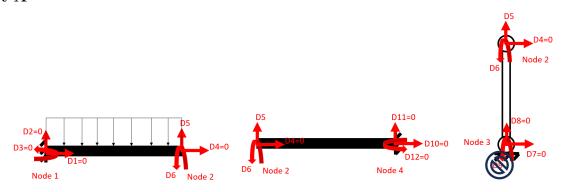


Figure 1

Part A



	Global node corresponding to		Coordinates in global		Direction cosines	
Element #	local node 1	local node 2	X_i, Y_i	X_j, Y_j	l_{ij}	m_{ij}
1	1	2	0, 0	1, 0	1	0
2	2	4	1, 0	2, 0	1	0
3	2	3	1, 0	1, -1	0	-1

$$k_{e_{beam}} = \frac{EI_z}{2a^3} \begin{bmatrix} 3 & 3a & -3 & 3a \\ & 4a^2 & -3a & 2a^2 \\ & & 3 & -3a \\ sym. & & 4a^2 \end{bmatrix}, \quad \frac{l_e}{2} = a, \quad a = 0.5, \quad I_z = \frac{\pi}{2}r^4 = \frac{\pi}{2}(\frac{A}{\pi})^2 = 1.59 * 10^{-5}$$

$$k_1 = k_2 = 4EI_z \begin{bmatrix} 3 & 1.5 & -3 & 1.5 \\ & 1 & -1.5 & 0.5 \\ & & 3 & -1.5 \\ sym. & & 1 \end{bmatrix}$$

$$d_1 = \begin{bmatrix} D_2 \\ D_3 \\ D_5 \\ D_6 \end{bmatrix}, \quad d_2 = \begin{bmatrix} D_5 \\ D_6 \\ D_{11} \\ D_{12} \end{bmatrix}$$

$$T = \begin{bmatrix} l_x & m_x & 0 & 0 & 0 & 0 \\ l_y & m_y & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & l_x & m_x & 0 \\ 0 & 0 & 0 & l_y & m_y & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 & 0 & 0 & 0 \\ 0 & 1 & 0 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 & 0 & 0 \\ 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \end{bmatrix}, \text{ T is identity matrix for both element 1 and 2:}$$

$$K_1 = K_2 = k_1 = k_2$$

Expand Global K Matrices

Combine the Global K Matrices

Define Other Global Matrices

$$D = \begin{bmatrix} D_1 \\ D_2 \\ D_3 \\ D_4 \\ D_5 \\ D_6 \\ D_7 \\ D_8 \\ D_{10} \\ D_{11} \\ D_{12} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \\ D_5 \\ D_6 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

$$f_{e} = \begin{bmatrix} f_{x}a + f_{sx1} \\ f_{y}a + f_{sy1} \\ f_{y}\frac{a^{2}}{3} + m_{s1} \\ f_{x}a + f_{sx2} \\ f_{y}a + f_{sy2} \\ -f_{y}\frac{a^{2}}{3} + m_{s2} \end{bmatrix}, \quad f_{1} = \begin{bmatrix} 0 + 0 \\ -4000(0.5) + R_{y1} \\ -4000\frac{(0.5)^{2}}{3} + M_{1} \\ 0 + 0 \\ -4000(0.5) + R_{y2} \\ 4000\frac{(0.5)^{2}}{3} + M_{2} \end{bmatrix} = \begin{bmatrix} 0 \\ -2000 + R_{y1} \\ -333.33 + M_{1} \\ 0 \\ 0 \\ 333.33 + M_{2} \end{bmatrix}$$

$$f_2 = \begin{bmatrix} 0+0\\ -4000/4\\ -M_2\\ 0+0\\ R_{y3}\\ M_3 \end{bmatrix} = \begin{bmatrix} 0\\ -500\\ -M_2\\ 0\\ R_{y3}\\ M_3 \end{bmatrix}$$

$$f_3 = \begin{bmatrix} -4000/4 \\ 0 \\ R_{y4} \\ 0 \end{bmatrix} = \begin{bmatrix} -500 \\ 0 \\ R_{y4} \\ 0 \end{bmatrix}$$

$$F = \begin{bmatrix} 0 \\ -2000 + R_{y1} \\ -333.33 + M_1 \\ 0 \\ -2000 \\ 333.33 \\ 0 \\ R_{y4} \\ 0 \\ R_{y3} \\ M_3 \end{bmatrix}$$

Solve KD=F for D_5 and D_6

using matlab:

$$D_5 = -1.7194 * 10^{-5} \text{ m}$$
 $D_6 = 1.31026 * 10^{-5} \text{ rad}$

Part B

from KD=F:

$$-2000 + R_{y1} = (-3.816D_5 + 1.908D_6) * 10^7$$

$$\boxed{R_{y1} = 2902.3 \text{ N}}$$

$$R_{y3} = (-3.816D_5 - 1.908D_6) * 10^7$$

$$R_{y3} = 402.31 \text{ N}$$

$$R_{y4} = (-4D_5) * 10^7$$
$$R_{y4} = 683.76 \text{ N}$$

Part C

sum reaction forces in truss:

$$\begin{aligned} F_t &= \sqrt{R_{x4}^2 + R_{y4}^2} = \sqrt{0^2 + 683.76^2} \\ \hline \left[F_t &= 683.76 \text{ N} \right] \end{aligned}$$