AEEM4058 - Homework 4

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Problem 1

Figure 1 shows a truss structure with two uniform members made of same material. The truss structure is constrained at two ends. The cross-sectional area of all the truss members is 0.01 m^2 , and the Young's modulus of the material is $2.0E10 \text{ N/m}^2$. Using the finite element method, calculate

- (a) all the nodal displacements;
- (b) the internal forces in all the truss members; and
- (c) the reaction forces at the supports.

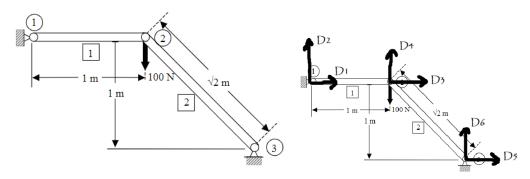


Figure 1

Part A

	Global node corresponding to		Coordinates in global		Direction cosines	
Element #	local node 1	local node 2	X_i, Y_i	X_j, Y_j	l_{ij}	m_{ij}
1	1	2	0, 0	1, 0	1	0
2	2	3	1, 0	2, -1	$\frac{1}{\sqrt{2}}$	$\frac{-1}{\sqrt{2}}$

Build Element Matrices

$$K_{e} = \frac{AE}{l_{e}} \begin{bmatrix} l_{ij}^{2} & l_{ij}m_{ij} & -l_{ij}^{2} & -l_{ij}m_{ij} \\ & m_{ij}^{2} & -l_{ij}m_{ij} & -m_{ij}^{2} \\ & & l_{ij}^{2} & l_{ij}m_{ij} \\ sym. & & & m_{ij}^{2} \end{bmatrix}$$

$$K_{e1} = \frac{0.01(2E10)}{1} \begin{bmatrix} 1 & 0 & -1 & 0 \\ & 0 & 0 & 0 \\ & 1 & 0 \\ sym. & & 0 \end{bmatrix} = \begin{bmatrix} 2 & 0 & -2 & 0 \\ & 0 & 0 & 0 \\ & 2 & 0 \\ sym. & & 0 \end{bmatrix} * 10^{8} \text{ N/m}$$

$$K_{e2} = \frac{0.01(2E10)}{\sqrt{2}} \begin{bmatrix} 0.5 & -0.5 & -0.5 & 0.5 \\ & 0.5 & 0.5 & -0.5 \\ & & 0.5 & -0.5 \\ sym. & & 0.5 \end{bmatrix} = \begin{bmatrix} 1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2} \\ & 1/\sqrt{2} & -1/\sqrt{2} \\ & & 1/\sqrt{2} & -1/\sqrt{2} \\ sym. & & & 1/\sqrt{2} \end{bmatrix} * 10^{8} \text{ N/m}$$

Build Global Matrices

combine K_{e1} and K_{e2} :

1

set up F and D matrices from boundary conditions:

$$F = \begin{bmatrix} R_{D1} \\ R_{D2} \\ 0 \\ -100 \\ R_{D5} \\ R_{D6} \end{bmatrix}, \quad D = \begin{bmatrix} 0 \\ 0 \\ D3 \\ D4 \\ 0 \\ 0 \end{bmatrix}$$

Solve for displacements

$$KD = F$$

$$10^{8} * \begin{bmatrix} 2 & 0 & -2 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ -2 & 0 & 2 + 1/\sqrt{2} & -1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2} \\ 0 & 0 & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} & -1/\sqrt{2} \\ 0 & 0 & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} & -1/\sqrt{2} \\ 0 & 0 & 1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ D3 \\ D4 \\ 0 \\ 0 \end{bmatrix} = \begin{bmatrix} R_{D1} \\ R_{D2} \\ 0 \\ -100 \\ R_{D5} \\ R_{D6} \end{bmatrix}$$

use Matlab to solve for displacements at node 2 (D3 and D4):

$$\begin{bmatrix} -2 & 0 & 2+1/\sqrt{2} & -1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2} \\ 0 & 0 & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} & -1/\sqrt{2} \end{bmatrix} \begin{bmatrix} D3 \\ D4 \end{bmatrix} = \begin{bmatrix} 0 \\ -100 \end{bmatrix}$$

$$\begin{bmatrix} 0 \\ 0 \\ -5*10^{-7} \\ -1.9142*10^{-6} \\ 0 \\ 0 \end{bmatrix}$$
 m

Part B

$$F_e = \frac{AE}{l_e} \begin{bmatrix} -l_{ij} & -m_{ij} & l_{ij} & m_{ij} \end{bmatrix} \begin{bmatrix} D1 \\ D2 \\ D3 \\ D4 \end{bmatrix}$$

$$F_1 = \frac{0.01(2E10)}{1} \begin{bmatrix} -1 & 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ -5 * 10^{-7} \\ -1.9142 * 10^{-6} \end{bmatrix}$$

solve with matlab for F:

$$F_1 = -100 \text{ N}$$

$$F_2 = \frac{0.01(2E10)}{\sqrt{2}} \begin{bmatrix} -\frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & \frac{1}{\sqrt{2}} & -\frac{1}{\sqrt{2}} \end{bmatrix} \begin{bmatrix} -5*10^{-7} \\ -1.9142*10^{-6} \\ 0 \\ 0 \end{bmatrix}$$

solve with matlab for F:

$$F_2 = -141.42 \text{ N}$$

Part C

plug in known values for displacements into KD=F:

plug in known values for displacements into KD=F:
$$10^8*\begin{bmatrix} 2 & 0 & -2 & 0 & 0 & 0\\ 0 & 0 & 0 & 0 & 0 & 0\\ -2 & 0 & 2+1/\sqrt{2} & -1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2}\\ 0 & 0 & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} & -1/\sqrt{2}\\ 0 & 0 & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} & -1/\sqrt{2}\\ 0 & 0 & 1/\sqrt{2} & -1/\sqrt{2} & 1/\sqrt{2} & 1/\sqrt{2} \end{bmatrix} \begin{bmatrix} 0\\0\\-5*10^{-7}\\-1.9142*10^{-6}\\0\\0\\0\end{bmatrix} = \begin{bmatrix} R_{D1}\\R_{D2}\\0\\-100\\R_{D5}\\R_{D6} \end{bmatrix}$$

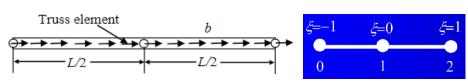
solve for each missing force in F with matlab:

$$\begin{bmatrix} R_{D1} \\ R_{D2} \\ R_{D5} \\ R_{D6} \end{bmatrix} = \begin{bmatrix} 100 \\ 0 \\ -100 \\ 100 \end{bmatrix}$$
N

Problem 2

Figure 2 shows a three-node truss element of length L and a constant cross-section area A. It is made of a material of Young's modulus E and density ρ . The truss is subjected to a uniformly distributed force b.

- (a) Derive the stiffness matrix for the element.
- (b) Write down the expression for the element mass matrix, and obtain m_{11} in terms of L, E, ρ , and A.
- (c) Derive the external force vector.



Part A

$$\begin{split} N_0(\xi) &= -\frac{1}{2}\xi(1-\xi) = -\frac{1}{2}\xi + \frac{1}{2}\xi^2 \\ N_1(\xi) &= (1+\xi)(1-\xi) = 1-\xi^2 \\ N_2(\xi) &= \frac{1}{2}\xi(1+\xi) = \frac{1}{2}\xi + \frac{1}{2}\xi^2 \\ a\xi &= x \to d\xi = \frac{1}{a}dx \to \frac{d\xi}{dx} = \frac{1}{a} \\ K &= \int_V B^T c B d V = A \int_{-L/2}^{L/2} B^T E B d x \\ B &= \frac{dN}{dx} = \frac{dN}{d\xi} \frac{d\xi}{dx} = \frac{dN}{d\xi} \frac{1}{a} \\ B &= \frac{1}{dx} \begin{bmatrix} dN_1 & dN_2 & dN_3 \end{bmatrix} \\ \frac{dN_1}{d\xi} &= -\frac{1}{2} + \xi, \quad \frac{dN_2}{d\xi} = -2\xi, \quad \frac{dN_3}{d\xi} = \frac{1}{2} + \xi \\ B &= \frac{1}{a} \begin{bmatrix} -\frac{1}{2} + \xi & -2\xi & \frac{1}{2} + \xi \end{bmatrix} \end{split}$$

$$K = EA \int_{-1}^{1} B^{T} Bad\xi$$

$$B^TB = \frac{1}{a^2} \begin{bmatrix} -\frac{1}{2} + \xi & -2\xi & \frac{1}{2} + \xi \end{bmatrix} \begin{bmatrix} -\frac{1}{2} + \xi \\ -2\xi \\ \frac{1}{2} + \xi \end{bmatrix} = \frac{1}{a^2} \begin{bmatrix} \frac{1}{4} - \xi + \xi^2 & \xi - 2\xi^2 & -\frac{1}{4} + \xi^2 \\ & 4\xi^2 & -\xi - 2\xi^2 \\ sym. & \frac{1}{4} + \xi + \xi^2 \end{bmatrix}$$

integrate and ignore the odd power terms of ξ since they will cancel:

$$K = \frac{EA}{a} \int_{-1}^{1} \begin{bmatrix} \frac{1}{4} - \xi + \xi^{2} & \xi - 2\xi^{2} & -\frac{1}{4} + \xi^{2} \\ 4\xi^{2} & -\xi - 2\xi^{2} \end{bmatrix} d\xi = \frac{EA}{a} \begin{bmatrix} \frac{1}{4}\xi + \frac{1}{3}\xi^{3} & -\frac{2}{3}\xi^{3} & -\frac{1}{4}\xi + \frac{1}{3}\xi^{3} \\ \frac{4}{3}\xi^{3} & -\frac{2}{3}\xi^{3} \end{bmatrix}_{-1}^{1}$$

$$K = \frac{2EA}{a} \begin{bmatrix} \frac{1}{4}\xi + \frac{1}{3}\xi^3 & -\frac{2}{3}\xi^3 & -\frac{1}{4}\xi + \frac{1}{3}\xi^3 \\ & \frac{4}{3}\xi^3 & -\frac{2}{3}\xi^3 \\ sym. & \frac{1}{4}\xi + \frac{1}{3}\xi^3 \end{bmatrix}_0^1 = \frac{2EA}{a} \begin{bmatrix} \frac{1}{4} + \frac{1}{3} & -\frac{2}{3} & -\frac{1}{4} + \frac{1}{3} \\ & \frac{4}{3} & -\frac{2}{3} \\ sym. & \frac{1}{4} + \frac{1}{3} \end{bmatrix}$$

$$K = \frac{EA}{L} \begin{bmatrix} \frac{7}{3} & -\frac{8}{3} & \frac{1}{3} \\ \frac{16}{3} & -\frac{8}{3} & \frac{1}{3} \\ sym. & \frac{7}{3} \end{bmatrix}$$

$$\begin{split} m_e &= \int_{V_e} \rho N^T N dV = A \rho a \int_{-1}^1 N^T N d\xi \\ N^T N &= \begin{bmatrix} -\frac{1}{2} \xi + \frac{1}{2} \xi^2 \\ 1 - \xi^2 \\ \frac{1}{2} \xi + \frac{1}{2} \xi^2 \end{bmatrix} \begin{bmatrix} -\frac{1}{2} \xi + \frac{1}{2} \xi^2 & 1 - \xi^2 & \frac{1}{2} \xi + \frac{1}{2} \xi^2 \end{bmatrix} = \\ \begin{bmatrix} \frac{1}{4} \xi^2 - \frac{1}{2} \xi^3 + \frac{1}{4} \xi^4 & -\frac{1}{2} \xi + \frac{1}{2} \xi^2 + \frac{1}{3} \xi^3 - \frac{1}{2} \xi^4 & -\frac{1}{4} \xi^2 + \frac{1}{4} \xi^4 \\ 1 - 2 \xi^2 + \xi^4 & \frac{1}{2} \xi + \frac{1}{2} \xi^2 - \frac{1}{2} \xi^3 - \frac{1}{2} \xi^4 \\ sym. & \frac{1}{4} \xi^2 + \frac{1}{2} \xi^3 + \frac{1}{4} \xi^4 \end{bmatrix} \end{split}$$

integrate and ignore the odd power terms of ξ since they will cance

$$\begin{split} m_e &= A\rho a \int_{-1}^1 \begin{bmatrix} \frac{1}{4}\xi^2 - \frac{1}{2}\xi^3 + \frac{1}{4}\xi^4 & -\frac{1}{2}\xi + \frac{1}{2}\xi^2 + \frac{1}{3}\xi^3 - \frac{1}{2}\xi^4 & -\frac{1}{4}\xi^2 + \frac{1}{4}\xi^4 \\ & 1 - 2\xi^2 + \xi^4 & \frac{1}{2}\xi + \frac{1}{2}\xi^2 - \frac{1}{2}\xi^3 - \frac{1}{2}\xi^4 \end{bmatrix} d\xi = \\ sym. & \frac{1}{4}\xi^2 + \frac{1}{2}\xi^3 + \frac{1}{4}\xi^4 \end{bmatrix} d\xi = \\ A\rho a \begin{bmatrix} \frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 & \frac{1}{6}\xi^3 - \frac{1}{10}\xi^5 & -\frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 \\ & \xi - \frac{2}{3}\xi^3 + \frac{1}{5}\xi^5 & \frac{1}{6}\xi^3 - \frac{1}{10}\xi^5 \\ sym. & \frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 \end{bmatrix}_{-1}^1 \end{split}$$

$$m_e = 2A\rho a \begin{bmatrix} \frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 & \frac{1}{6}\xi^3 - \frac{1}{10}\xi^5 & -\frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 \\ \xi - \frac{2}{3}\xi^3 + \frac{1}{5}\xi^5 & \frac{1}{6}\xi^3 - \frac{1}{10}\xi^5 \\ sym. & \frac{1}{12}\xi^3 + \frac{1}{20}\xi^5 \end{bmatrix}^1 = 2A\rho a \begin{bmatrix} \frac{1}{12} + \frac{1}{20} & \frac{1}{6} - \frac{1}{10} & -\frac{1}{12} + \frac{1}{20} \\ 1 - \frac{2}{3} + \frac{1}{5} & \frac{1}{6} - \frac{1}{10} \\ sym. & \frac{1}{12} + \frac{1}{20} \end{bmatrix}$$

then plug in
$$a = L/2$$
:
$$m_e = A\rho L \begin{bmatrix} \frac{2}{15} & \frac{1}{15} & -\frac{1}{30} \\ \frac{8}{15} & \frac{1}{15} \\ sym. & \frac{2}{15} \end{bmatrix}$$

find m_{11} :

$$m_{11} = \frac{2A\rho L}{15}$$

Part C

$$f_{e} = \int_{V_{e}} N^{T} f_{b} dV + \int_{S_{e}} N^{T} f_{s} dS = A \int_{-1}^{1} N^{T} f_{b} a d\xi + L \int_{-1}^{1} N^{T} f_{s} a d\xi$$

$$f_{e} = (Aaf_{b} + Laf_{s}) \int_{-1}^{1} N^{T} d\xi = (Aaf_{b} + Laf_{s}) \int_{-1}^{1} \begin{bmatrix} -\frac{1}{2}\xi + \frac{1}{2}\xi^{2} \\ 1 - \xi^{2} \\ \frac{1}{2}\xi + \frac{1}{2}\xi^{2} \end{bmatrix} d\xi =$$

$$(Aaf_{b} + Laf_{s}) \begin{bmatrix} -\frac{1}{4}\xi^{2} + \frac{1}{6}\xi^{3} \\ \xi + \frac{1}{3}\xi^{3} \\ \frac{1}{4}\xi^{2} + \frac{1}{6}\xi^{3} \end{bmatrix}_{-1}^{1} = (Aaf_{b} + Laf_{s}) \begin{bmatrix} -\frac{1}{12} + \frac{5}{12} \\ \frac{2}{3} + \frac{2}{3} \\ \frac{1}{12} - \frac{1}{12} \end{bmatrix}$$

$$f_{e} = (Aaf_{b} + Laf_{s}) \begin{bmatrix} \frac{1}{3} \\ \frac{4}{3} \\ \frac{1}{3} \end{bmatrix}$$