

Parameter estimation

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ARTICLE INFO

Keywords:

Supercritical extraction
Parameter estimation
Mathematical modeling

ABSTRACT

We consider a system of partial differential equations, $G(t, x, \dot{x}, \Theta, u) = 0$, where x denotes the state variables, Θ represents the parameters, and u is the control variables. Our focus is on the supercritical extraction process, which involves a partially filled extractor with a fixed bed that operates under constant conditions. We assume the flow is uniform across any cross-section, although the area available for the fluid phase can vary along the extractor. To model this two-dimensional case, we use the concept of quasi-one-dimensional flow. To describe the fluid-solid extraction process of caraway oil from caraway seeds with CO_2 as a solvent, we use a distributed-parameter model based on Reverchon [1]. This model requires parameters such as partition factor, internal diffusion coefficient, axial diffusion coefficient, and an initial state estimate. We estimate these parameters by applying the maximum likelihood estimation method on yield data under the normal error assumption. The dataset comes from four experiments conducted at different operating conditions: 40°C/200 bar, 50°C/200 bar, 40°C/300 bar, and 50°C/300 bar.

1. Introduction

Over the years, the extraction of natural substances from solid materials and liquids using solvents has been a topic of significant interest for research and development. Supercritical fluids are particularly useful in extraction processes due to their pressure-dependent dissolving power and ability to exhibit both gas- and liquid-like properties, such as fluid-like density and gas-like diffusivity. Of the various supercritical fluids available, supercritical CO_2 is particularly attractive due to its nontoxic, non-flammable, and non-corrosive properties. Moreover, the critical point of CO_2 is relatively low (73.8 bar and 31°C) compared to other fluids, making it an excellent alternative to traditional extraction techniques.

Different mathematical models have been proposed to describe the extraction of valuable compounds from a fixed biomass bed, as reviewed by Huang et al. [2]. However, the selection of an appropriate extraction model should be based on the knowledge of the physical phenomena taking place in the operational unit. Each model has its own set of assumptions and describes different mass transfer mechanisms and equilibrium relationships.

Based on an analogy to heat transfer, Reverchon et al. [3] proposed a hot ball model to describe an extraction process from solid particles containing small quantities of solute, where solubility is not a limiting factor. Sovova [4] presented the Broken-and-Intact Cell model, which describes a system where the outer surfaces of particles have been mechanically interrupted, allowing easy access of solvent to the solute from the broken cells. In contrast, the solute from the intact cells is less accessible due to high mass transfer resistance.

Reverchon [1] developed a model for fluid-solid extraction, where the oil is treated as a single component,

and the extraction process is controlled by internal mass transfer resistance, neglecting external mass transfer. This model, however, does not consider the influence of axial dispersion or changes in density and flow rate along the bed. Our work assumes that the extraction process operates semi-continuously in a cylindrical vessel. The solvent is first brought to supercritical conditions, pumped through a fixed bed of finely chopped biomass, and the solute is extracted from the biomass. The solvent and solute are then separated in a flush drum, and the extract is collected. The feed flow rate (F_{in}) and inlet temperature (T_{in}) of the extractor can be measured and manipulated. In contrast, the vessel pressure (P) can be measured and manipulated, but the outlet temperature (T_{out}) can only be measured. Figure 1 depicts a simplified flow diagram.

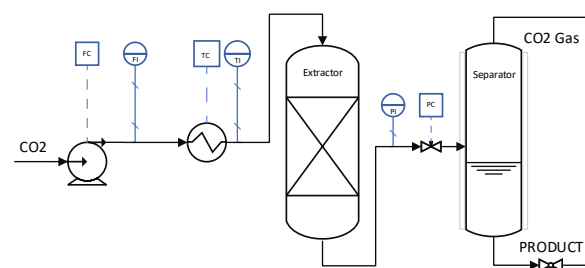


Figure 1: Process flow diagram (Font size)

2. Materials and methods

2.1. Supercritical fluids

A supercritical fluid (SCF) is a substance at a temperature and pressure above its critical point, where there are no distinct liquid and gas phases but below the pressure required to compress it into a solid. SCFs can move through porous solids like gases, which is faster than liquid transport through such materials. SCFs have a higher ability to dissolve materials like liquids or solids compared to gases. Near the

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critical point, small changes in pressure or temperature result in significant changes in density, allowing many properties of an SCF to be fine-tuned. By changing the pressure and temperature, the properties can be tuned to be more liquid-like or gas-like.

Fluid properties can be divided into two kinds: equilibrium properties and transport properties. The equation of state can be used accurately to predict the equilibrium properties, such as density, enthalpy, vapour pressure, fugacity and fugacity coefficient, vapour-liquid equilibrium, and all kinds of excess properties.

Supercritical CO_2 's thermodynamic properties, such as density, the local speed of sound, and specific heat capacity, vary significantly with slight changes in temperature and pressure due to real gas effects. The Peng-Robinson equation of state (P-R EOS) is used to calculate the thermodynamic properties by accounting for these real gas effects. The P-R EOS belongs to a specific class of thermodynamic models for modelling the pressure of a gas as a function of temperature and density. It can be written as a cubic function of the molar volume (of the density). Detail information about Peng-Robinson equation of state can be found in the work of Peng and Robinson [5], Elliott [6] or Pratt [7]. The P-R EOS is presented by equation 1

$$P = \frac{RT}{V_m - b} - \frac{a\alpha}{V_m^2 + 2bV_m - b^2} \quad (1)$$

The parameters a , b , α are parameters defined as presented in the appendix A.1.1.

The properties of CO_2 are presented as a function of operating conditions (temperature and pressure) in Figure 2. At standard atmospheric pressure and temperature, CO_2 behaves as an ideal gas, and its compressibility factor equals unity. However, at high pressures and/or low temperatures, intermolecular forces between gas molecules become more significant, causing them to deviate from ideal behaviour. As a result, the compressibility factor can either be greater than or less than unity, depending on the magnitude of these forces. As presented in Figure 2a, the compressibility factor obtained from the Peng-Robinson equation of state varies strongly depending on the operating conditions. The compressibility factor can be obtained by solving the polynomial form of the P-R EOS given by 2.

$$Z^3 - (1 - B)Z^2 + (A - 2B - 3B^2)Z - (AB - B^2 - B^3) = 0 \quad (2)$$

where A and B are parameters as defined in the appendix A.1.1. The roots of the polynomial can be found iteratively or by the Cardano formula. In a one-phase region, the fluid is described by one real root corresponding to the gas, liquid or supercritical phase. The gas-liquid mixture is present in the two-phase region, and two roots are found. The biggest root is assigned to the gas phase, and the smallest root corresponds to the liquid phase.

The real gas effects are also visible on the density plot presented in Figure 2b. The density calculations are based on the compressibility factor and its value depends on the operating conditions. The fluid's properties near the critical point are unique and combine gas-like and liquid-like

properties. The details of calculations are explained in the appendix A.1.2.

Figure 2c show the behaviour of the heat capacity of a supercritical fluid at constant pressure (C_p). The details of the calculations can be found in the appendix A.1.3. Contrary to the density, which varies monotonically, the specific heat shows very high levels in a narrow region. In the subcritical region, the phase transition is associated with an effective spike in the heat capacity (i.e., the latent heat). Approaching the critical point, the latent heat falls to zero, which is accompanied by a gradual rise in heat capacity in the pure phases near phase transition. At the critical point, the latent heat is zero, but the heat capacity shows a diverging singularity. Beyond the critical point, there is no divergence, but rather a smooth peak in the heat capacity; the highest point of this peak identifies the Widom line (as discussed by Simeoni et al. [8] and Banuti [9]).

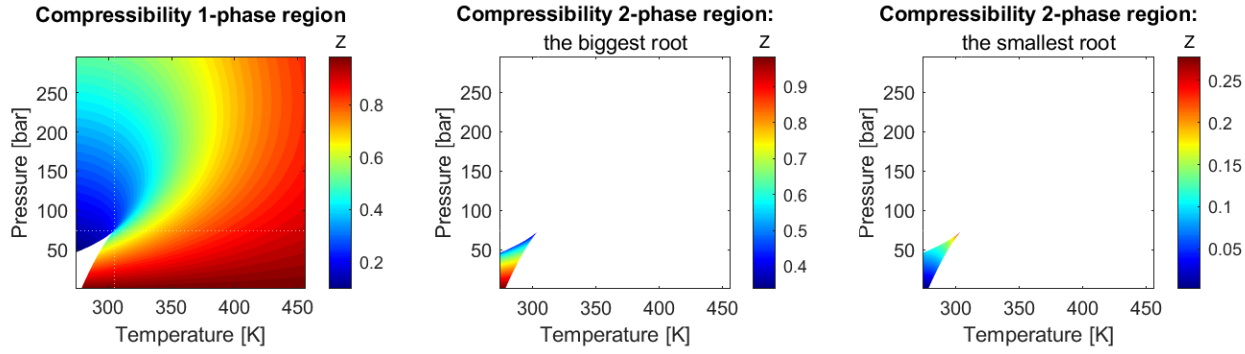
To determine the thermodynamic properties of a real gas, it is necessary to evaluate the departure function of the chosen equation of state for that property. As explained by Elliott [6], the departure function is the difference between the actual value of a thermodynamic property of a real gas and its value if the gas were ideal under the same temperature and pressure conditions. The ideal gas serves as a reference state to which the properties of real gases are compared. The departure function is a measure of the extent to which a real gas deviates from ideal gas behaviour. The departure functions allow for the accurate calculation of thermodynamic properties for real gases.

Transport properties such as viscosity and conductivity play a crucial role in engineering design for production, fluid transportation, and processing. However, as highlighted by Sheng et al. [10], developing a satisfactory theory for transport properties of real dense gases and liquids is a challenging task. This is due to the inherent difficulties involved in accurate measurements and the complexity involved in theoretical treatments.

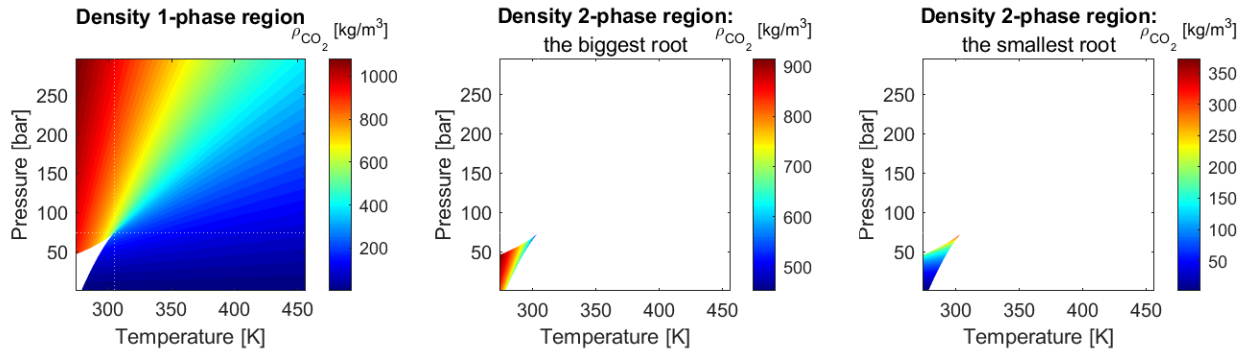
To address this issue, the correlations of transport coefficients are either empirical or based on some theoretical foundation. Chapman-Enskog's theory (presented in Chapman and Cowling [11]) for transport properties of dense gases based on the distribution function is a popular theoretical approach. However, the Chapman-Enskog theory was developed for rigid spherical molecules and modifications are required to apply it to real gases. Many correlations have been proposed following the Chapman-Enskog theory in the form of reduced density and reduced temperature, such as those developed by Fenghour et al. [12] and Laesecke and Muzny [13] from the National Institute of Standards and Technology (NIST). A comparison of these correlations is presented in Figure 3.

NIST has developed a viscosity formulation consisting of four contributions: (i) for the limit of zero density, (ii) for the initial density dependence, (iii) for the residual viscosity, and (iv) for the singularity of the viscosity at the critical point. The NIST correlation covers temperatures from 100 to 2000 K for gaseous CO_2 , and from 220 to 700 K with

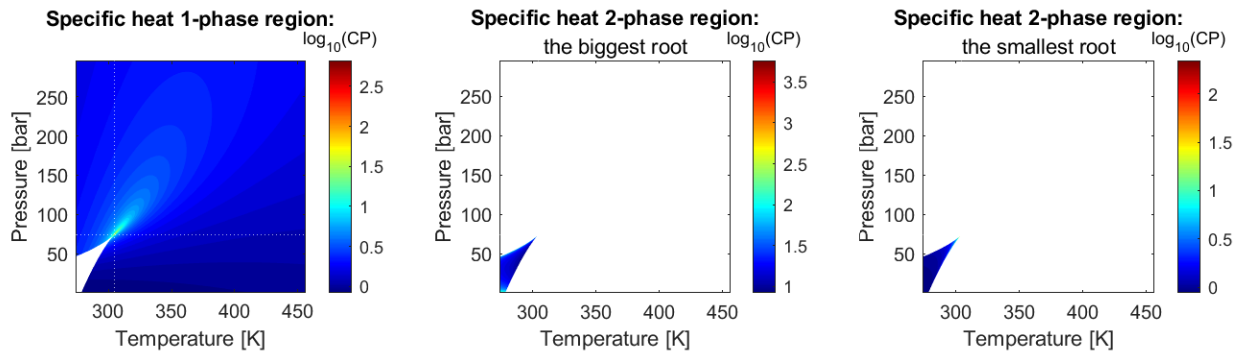
Parameter estimation



(a) The compressibility factor based on the Peng-Robinson equation of state



(b) The fluid density based on the Peng-Robinson equation of state



(c) The specific heat of the CO_2 based on the Peng-Robinson equation of state

Figure 2: Properties of CO_2 based on the equation of state

pressures along the melting line up to 8000 MPa for compressed and supercritical liquid states. These correlations and theories are essential in predicting transport properties for real gases and liquids and can assist in engineering design and analysis.

Similarly, several correlations for thermal conductivity of CO_2 were compared in Figure 4. The presented figures show regions around the critical point where the singularity

is present. Similarities between specific heat and thermal conductivity can be observed. The NIST correlation (Huber et al. [14]) captures the singular behaviour of thermal conductivity around the critical point. The correlations are applicable for the temperature range from the triple point to 1100 K and pressures up to 200 MPa .

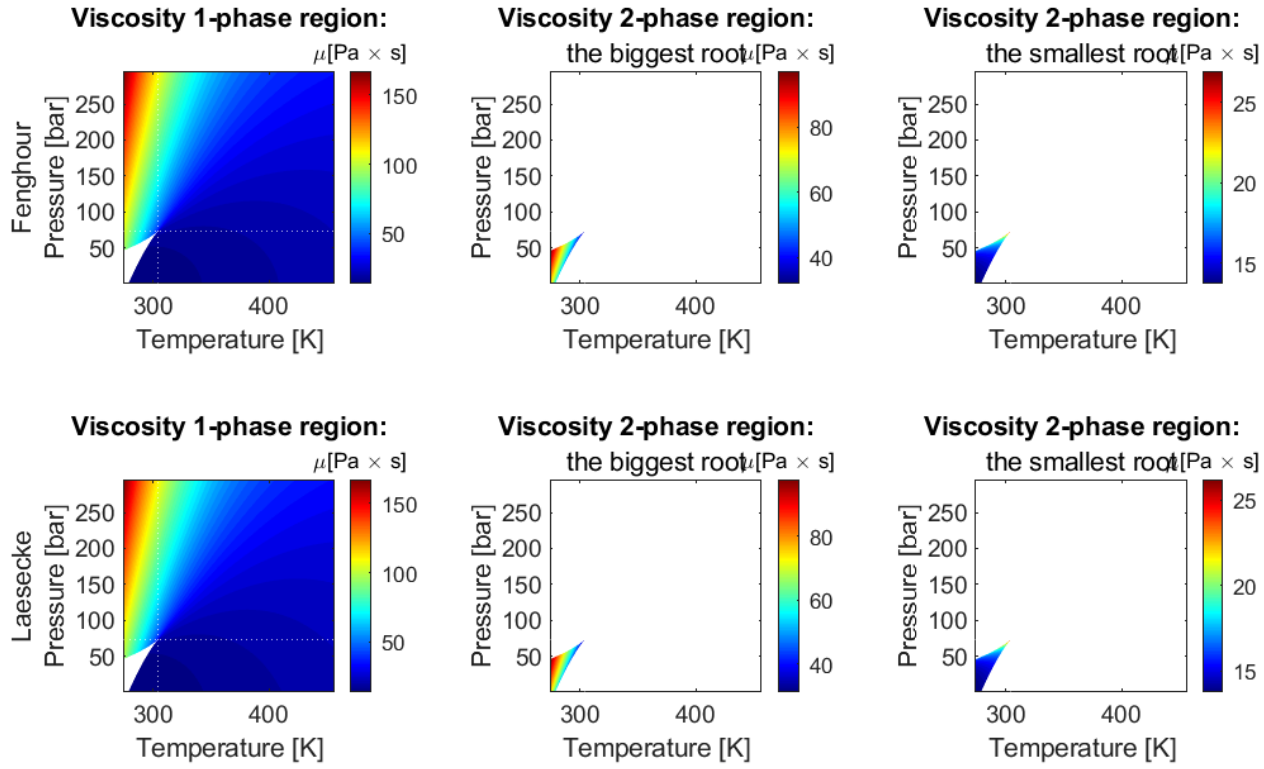


Figure 3: Viscosity obtained based on different correlations

2.2. Governing equations

The governing equation for a quasi-one-dimensional compressible flow in Cartesian coordinates can be found in the appendix A.2 and in the work of Anderson [15]. Quasi-one-dimensional flow is a fluid flow characterized by the assumption that the flow properties remain uniform across any given cross-section of the flow. This assumption is made when there is a variation in the cross-sectional area of the flow channel, such as an irregular shape or partial filling of an extractor. In such cases, the flow is considered to be quasi-one-dimensional because the velocity and other flow properties are assumed to vary only in the direction of flow.

The quasi-one-dimensional compressible Navier-Stokes equations in Cartesian coordinates are given by equations 3 to 5.

$$\frac{\partial(\rho_f A_f)}{\partial t} + \frac{\partial(\rho_f A_f v)}{\partial z} = 0 \quad (3)$$

$$\frac{\partial(\rho_f v A_f)}{\partial t} + \frac{\partial(\rho_f A_f v^2)}{\partial z} = -A_f \frac{\partial P}{\partial z} \quad (4)$$

$$\frac{\partial(\rho_f e A_f)}{\partial t} + \frac{\partial(\rho_f A_f v e)}{\partial z} = -P \frac{\partial(A_f v)}{\partial z} + \frac{\partial}{\partial z} \left(\frac{\partial T}{\partial z} \right) \quad (5)$$

where ρ_f is the density of the fluid, A_f is the function which describe change of the cross-section, v is the velocity, P is the total pressure, e is the internal energy of the fluid, t is time and z is the spacial direction.

Based on governing equations, the small discontinuity (defined as δ) in flow properties, shown in Figure 5, can be analysed. The analysis follows the work of Schreier [16].

The discontinuity is presumed to be at rest relative, and the balance equations become

$$\rho_f \delta v + v \delta \rho_f + \delta \rho_f \delta v = 0$$

$$\delta P = \delta v \delta \rho_f$$

These relations are equally valid if the two regions are separated by a region of finite width rather than a discontinuity.

$$\lim_{\rho_f v \rightarrow 0} \rho_f \delta v + v \delta \rho_f + \delta \rho_f \delta v = 0 / \delta \rho_f \rightarrow \frac{dv}{d\rho_f} = -\frac{v}{\rho_f}$$

By combining the momentum equation with the above equation, we get

$$\frac{dv}{d\rho_f} = -\frac{dv}{dP} \frac{dP}{d\rho_f} = -\frac{1}{\rho_f v} \frac{dP}{d\rho_f} = -\frac{v}{\rho_f}$$

Suppose the flow is presumed to be isentropic, $dP/d\rho = c^2$, so $v^2 = c^2$, where c is the speed of sound. This can be interpreted as a small pressure wave propagating with the speed of sound relative to the flow.

2.3. Low Mach number expansion

As discussed by Lions [17], the low Mach number equations are a subset of the fully compressible equations of motion (continuity, momentum and energy). Such a

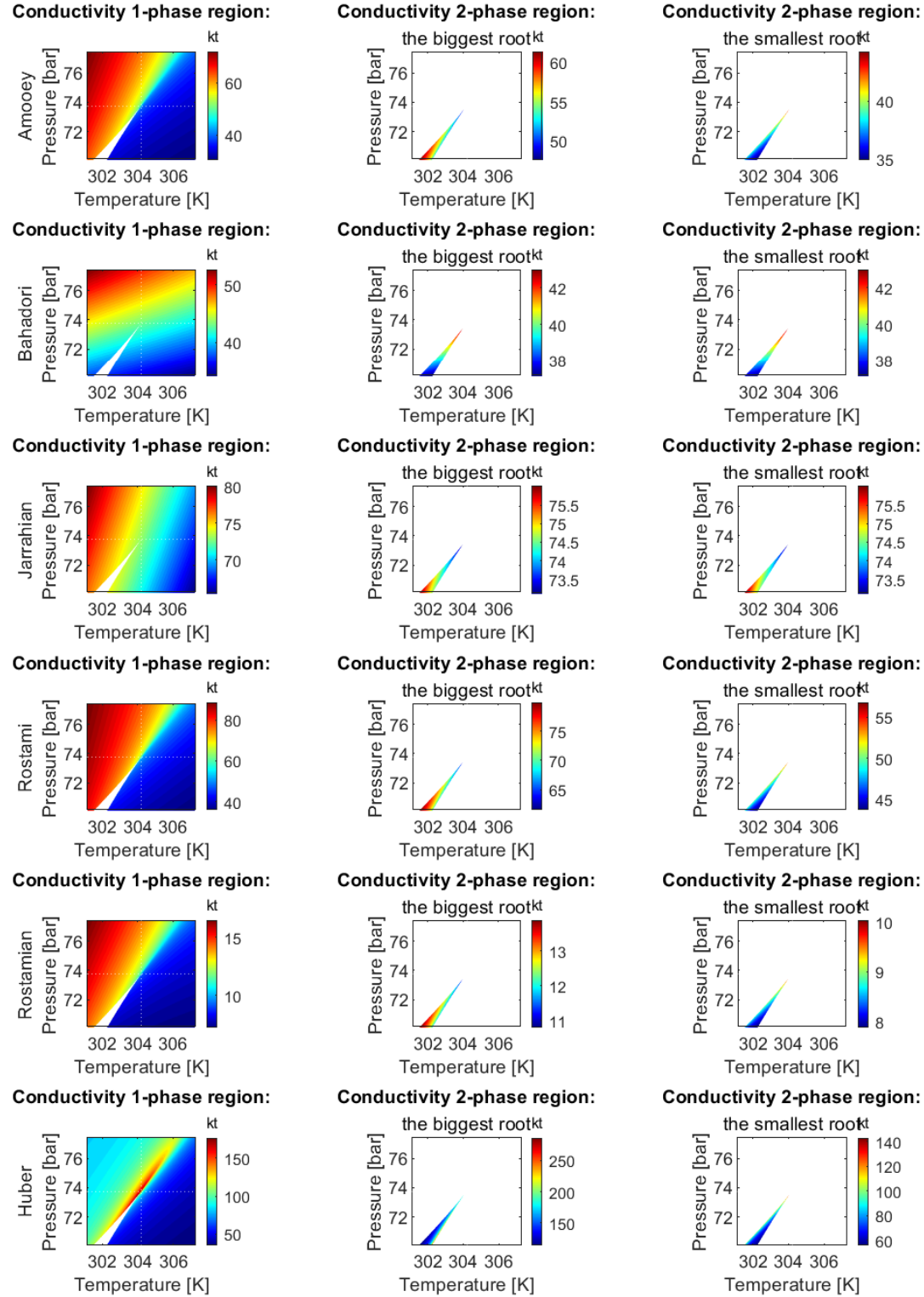


Figure 4: Thermal conductivity obtained based on different correlations

set of equations allow for large variations in gas density, but it is considered acoustically incompressible. Therefore, the low Mach number equations are preferred over the full compressible equations for low-speed flow problems

$\left(M_a = \frac{|v|}{\sqrt{\partial P / \partial \rho_f}} \ll 1 \right)$ to avoids the need to resolve fast-moving acoustic signals. The equations are derived from compressible equations based on the perturbation theory.

	ρ_f	$\rho_f + \delta\rho_f$	
$v \rightarrow$	P	$P + \delta P$	$v + \delta v \rightarrow$
	T	$T + \delta T$	

Figure 5: Small discontinuity in one-dimensional flow

The perturbation theory develops an expression for the desired solution in terms of a formal power series known as a perturbation series in some "small" parameter ζ , that quantifies the deviation from the exactly solvable problem. The leading term in this power series is the solution of the exactly solvable problem, while further terms describe the deviation in the solution due to the deviation from the initial problem.

The equations 3 to 4 describe the fully compressible equations of motion (respectively, the transport of mass, momentum and energy) for the quasi-one-dimensional case. We rescale the time variable, considering finally

$$\rho_\zeta = \rho(z, t/M_a), \quad v_\zeta = \frac{1}{\zeta} v(z, t/M_a)$$

$$T_\zeta = T(z, t/M_a), \quad k_\zeta = \zeta k(\rho_f, T)$$

The conservative non-dimensional equations of motion become

$$\frac{\partial(\rho_\zeta A_f)}{\partial t} + \frac{\partial(\rho_\zeta A_f v_\zeta)}{\partial z} = 0$$

$$\frac{\partial(\rho_\zeta A_f v_\zeta)}{\partial t} + \frac{\partial(\rho_\zeta v_\zeta A_f v_\zeta)}{\partial z} + \frac{A_f}{M_a^2} \frac{\partial P_\zeta}{\partial z} = 0$$

$$\frac{\partial(\rho_\zeta e_\zeta A_f)}{\partial t} + \frac{\partial(\rho_\zeta e_\zeta v_\zeta A_f)}{\partial z} - \frac{\partial}{\partial z} \left(k \frac{\partial T_\zeta}{\partial z} \right) + P_\zeta \frac{\partial(A_f v_\zeta)}{\partial z} = 0$$

Let's define $\zeta = M_a^2$ and assume small Mach numbers, $M_a \ll 1$, then the kinetic energy, viscous work, and gravity work terms can be neglected in the energy equation since the square of the Mach number scales those terms. The inverse of Mach number squared remains in the momentum equations, suggesting singular behaviour. In order to explore the singularity, the pressure, velocity and temperature are expanded as asymptotic series in terms of the parameter ζ

$$P_\zeta = P_0 + P_1 \zeta + P_2 \zeta^2 + \mathcal{O}(\zeta^3)$$

$$\rho_\zeta = \rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)$$

$$v_\zeta = v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)$$

$$T_\zeta = T_0 + T_1 \zeta + \mathcal{O}(\zeta^2)$$

$$e_\zeta = e_0 + e_1 \zeta + \mathcal{O}(\zeta^2)$$

By expanding performing power expansion on the continuity equation and taking the limit of ζ from the positive side, we get

$$\lim_{\zeta \rightarrow 0_+} \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f)}{\partial t} + \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial z} = 0$$

The continuity equation becomes

$$\frac{\partial(\rho_0 A_f)}{\partial t} + \frac{\partial(\rho_0 A_f v_0)}{\partial z} = 0 \quad (6)$$

The form of the continuity equation stays the same. Considering the momentum equation, it can be seen that the inverse of Mach number squared remains, which suggests singular behaviour.

$$\lim_{\zeta \rightarrow 0_+} \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial t} + \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)) (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial z} + A_f \frac{\partial}{\partial z} \left(\frac{P_0}{M_a^2} + \frac{P_1 \zeta}{M_a^2} + \frac{P_2 \zeta^2}{M_a^2} + \mathcal{O}(\zeta^3) \right)$$

The first two terms stay the same, but the third one becomes different in structure. By further investigation of the pressure term in the momentum equation, it can be observed.

$$\lim_{\zeta \rightarrow 0_+} \frac{\partial}{\partial z} \left(\frac{P_0}{M_a^2} + \frac{P_1 \zeta}{M_a^2} + \frac{P_2 \zeta^2}{M_a^2} + \mathcal{O}(\zeta^3) \right) =$$

$$= \lim_{\zeta \rightarrow 0_+} \frac{\partial}{\partial z} \left(\frac{P_0}{M_a^2} \right) + \frac{\partial}{\partial z} \left(\frac{P_1 \zeta^2}{M_a^2} \right) + \frac{\partial}{\partial z} \left(\frac{P_2 \zeta^2}{M_a^2} \right)$$

$$= \lim_{\zeta = M_a^2 \rightarrow 0_+} \frac{\partial}{\partial z} \left(\frac{P_0}{M_a^2} \right) + \frac{\partial}{\partial z} \left(\frac{P_1 M_a^2}{M_a^2} \right) + \frac{\partial}{\partial z} \left(\frac{P_2 M_a^4}{M_a^2} \right)$$

$$= \lim_{\zeta = M_a^2 \rightarrow 0_+} 0 + \frac{\partial P_1}{\partial z} + 0$$

The simplification of the P_0 in the momentum equation comes from the fact that P_0 is independent of z . As was presented above, the thermodynamical pressure moves with the speed of sound, and any perturbation propagates instantaneously. The term related to P_2 and higher order terms become zero at the limit of $M_a \rightarrow 0$. The momentum equation becomes

$$\frac{\partial(\rho_0 A_f v_0)}{\partial t} + \frac{\partial(\rho_0 v_0 A_f v_0)}{\partial z} + A_f \frac{\partial P_1}{\partial z} = 0$$

By expanding performing power expansion on the energy equation and taking the limit of ζ from the positive side, we get

$$\lim_{\zeta \rightarrow 0_+} \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f (e_0 + e_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial t} + \frac{\partial((\rho_0 + \rho_1 \zeta + \mathcal{O}(\zeta^2)) A_f (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)) (e_0 + e_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial z} + \frac{\partial}{\partial z} \left(k \frac{\partial}{\partial z} (T_0 + T_1 \zeta + \mathcal{O}(\zeta^2)) \right) - (P_0 + P_1 \zeta + P_2 \zeta^2 + \mathcal{O}(\zeta^3)) \frac{\partial(A_f (v_0 + v_1 \zeta + \mathcal{O}(\zeta^2)))}{\partial z} = 0$$

The form of the energy equation stays the same.

$$\frac{\partial(\rho_0 e_0 A_f)}{\partial t} + \frac{\partial(\rho_0 e_0 v_0 A_f)}{\partial z} - \frac{\partial}{\partial z} \left(k \frac{\partial T_0}{\partial z} \right) + P_0 \frac{\partial(A_f v_0)}{\partial z} = 0$$

where $e_0 = e(\rho_0, T_0)$ and $k = k(\rho_0, T_0)$.

The expansion results in two different types of pressure and they are considered to be split into a thermodynamic component (P_0) and a dynamic component (P_1). The thermodynamic pressure is constant in space, but can change in

time. The thermodynamic pressure is used in the equation of state. The dynamic pressure only arises as a gradient term in the momentum equation and acts to enforce continuity.

The resulting unscaled low Mach number equations are:

$$\begin{aligned} \frac{\partial (\rho_f A_f)}{\partial t} + \frac{\partial (\rho_f A_f v)}{\partial z} &= 0 \\ \frac{\partial (\rho_f A_f v)}{\partial t} + \frac{\partial (\rho_f v A_f v)}{\partial z} + A_f \frac{\partial P_1}{\partial z} &= 0 \\ \frac{\partial (\rho_f e A_f)}{\partial t} + \frac{\partial (\rho_f e v A_f)}{\partial z} - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} &= 0 \end{aligned}$$

The energy equation can be expanded through the chain rule to obtain

$$\rho A_f \left(\frac{\partial e}{\partial t} + v \frac{\partial e}{\partial z} \right) + e \underbrace{\left(\frac{\partial (\rho_f A_f)}{\partial t} + \frac{\partial (\rho_f v A_f)}{\partial z} \right)}_{\text{Continuity}} - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} = 0$$

The non-conservative form of the energy equation becomes

$$\rho A_f \left(\frac{\partial e}{\partial t} + v \frac{\partial e}{\partial z} \right) - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} = 0$$

If the calorically perfect gas is assumed, then $e = C_v T$, where C_v is the constant specific heat. The energy equation can be derived in terms of temperature T .

$$\rho A_f C_v \left(\frac{\partial T}{\partial t} + v \frac{\partial T}{\partial z} \right) - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} = 0$$

If an isothermal case is assumed, then the energy equation becomes

$$\lim_{\Delta T \rightarrow 0^+} \rho A_f C_v \left(\frac{\partial T}{\partial t} + v \frac{\partial T}{\partial z} \right) - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} = 0$$

which leads to

$$\frac{\partial (A_f v)}{\partial z} = 0 \quad (7)$$

In one-dimensional case, the equation 7 become equivalent of $\text{div}(A_f v) = 0$, which known as the incompressibility condition (Lions [17]).

As presented by Elliott [6], a general formulation of the internal energy for a real gas is:

$$de = C_v dT - \left[P - T \left(\frac{\partial P}{\partial T} \right)_{v_m} \right] dv_m$$

where v_m is the molar volume.

The internal energy is a function of two intensive properties, in this case, T and $v_m = 1/\rho_f$. But, in the case of an ideal gas, the equation of state is such that the second term in this equation is identically equal to zero. So the ideal gas is a special case in which the molar internal energy is a function only of temperature. For Peng-Robinson equation of state, the internal energy is defined as

$$e = C_v T + \frac{a \left(\alpha - T \frac{d\alpha}{dT} \right)}{2\sqrt{2}b} \ln \left[\frac{1 + b(1 - \sqrt{2}\rho)}{1 + b(1 + \sqrt{2}\rho)} \right]$$

Assuming constant temperature and pressure along space and in time:

$$\lim_{\Delta T, \Delta P \rightarrow 0^+} \rho A_f \left(\frac{\partial e}{\partial t} + v \frac{\partial e}{\partial z} \right) - \frac{\partial}{\partial z} \left(k \frac{\partial T}{\partial z} \right) + P_0 \frac{\partial (A_f v)}{\partial z} = 0 \rightarrow \frac{\partial (A_f v)}{\partial z} = 0$$

It can be deduced that the continuity equation becomes $\frac{\partial \rho_f}{\partial t} = 0$ at constant temperature and pressure. Moreover, the incompressibility condition $\text{div}(A_f v) = 0$ is obtained.

Assuming an arbitrary function \hat{P} , which describes the total pressure, the dimensional momentum equation can be written as

$$v \underbrace{\left(\frac{\partial (\rho_f A_f)}{\partial t} + \frac{\partial (\rho_f v A_f)}{\partial z} \right)}_{\text{Continuity equation}} + \rho_f A_f \frac{\partial v}{\partial t} + \rho_f v A_f \frac{\partial v}{\partial z} = -A_f \frac{\partial \hat{P}}{\partial z}$$

From the incompressibility conditions, we can deduce that

$$\frac{\partial (A_f v)}{\partial z} = 0 \rightarrow A_f \frac{\partial v}{\partial z} = -v \frac{\partial A_f}{\partial z}$$

By combining both above equations, assuming that $\partial v / \partial t = 0$:

$$\frac{\rho_f v^2}{A_f} \frac{\partial A_f}{\partial z} = \frac{\partial \hat{P}}{\partial z} \rightarrow \int \frac{\rho_f v^2}{A_f} \frac{\partial A_f}{\partial z} dz = \int \frac{\partial \hat{P}}{\partial z} dz$$

The l.h.s integral can be solved by assuming ρ_f is constant and introducing superficial velocity $u_s = A_f v$

$$\begin{aligned} \int \frac{\rho_f v^2}{A_f} \frac{\partial A_f}{\partial z} dz &= \int \frac{\rho_f v^2}{A_f} \frac{A_f^2}{A_f^2} \frac{\partial A_f}{\partial z} dz \\ &= \rho_f u_s^2 \int \frac{1}{A_f^3} \frac{\partial A_f}{\partial z} dz = -\frac{\rho_f u_s^2}{2\Delta A_f^2} = -\frac{\rho_f \Delta v^2}{2} \\ \int \frac{\partial \hat{P}}{\partial z} dz &= \Delta \hat{P} \end{aligned}$$

The final form of the momentum equation corresponds to Bernoulli's principle

$$\Delta \hat{P} = -\frac{\rho_f \Delta v^2}{2} \xrightarrow{P_0 = \text{const}} \Delta M_a^2 P_1 = -\frac{\rho_f \Delta v^2}{2}$$

Bernoulli's principle can be used to find the hydrodynamic pressure caused by varying cross-sections at steady-state. Moreover, if the flow velocity is relatively low, all pressure changes are hydrodynamic (due to velocity motion) rather than thermodynamic. The effect of this is that $\partial \rho / \partial P = 0$. In other words, the small changes in pressure due to flow velocity changes do not change the density. This has a secondary effect – the speed of sound in the fluid is $\partial P / \partial \rho = \infty$ in this instance. So there is an infinite speed of sound, which makes the equations elliptic in nature. It can be deduced that at the isothermal conditions, the density in the system propagates with the same speed as pressure since they are both connected through the equation of state.

2.4. Extraction model

For the sake of clarity of the process model, different colors have been used in the equations to indicate: **control variables**, **state variables**, **variables** and **parameters**.

2.4.1. Continuity equation

The continuity equation for the fluid phase is derived in Appendix A.2. When the cross-sectional area of the channel A_f is specified as a function of the void fraction $\phi(z)$ (where ϕ is the void fraction of the bed and A is the cross-section of the empty extractor), the continuity equation takes the form:

assign a color of v and u

$$\frac{\partial(\rho_f(T(t, z), P(t))\phi)}{\partial t} + \frac{\partial(\rho_f(T(t, z), P(t))vA\phi)}{\partial z} = 0$$

Assuming that the mass flow rate is constant in time, the temporal derivative becomes zero, and the spatial derivative can be integrated along z as

$$\int \frac{\partial(\rho_f(T(t, z), P(t))vA\phi)}{\partial z} dz = 0 \rightarrow F = \rho_f(T(t, z), P(t))vA\phi \quad (8)$$

Here, F is a constant obtained from the integration and is understood as the mass flux per unit area, which is assumed to be constant along z . To simplify the dynamics of the system, it is assumed that $F = F(t)$ is a control variable and affects the whole system instantaneously. This assumption allows for finding the velocity profile that satisfies mass continuity based on $F(t)$, $\phi(z)$, and $\rho_f(T(t, z), P(t))$.

$$v = \frac{F(t)}{\rho_f(T(t, z), P(t))A\phi} \quad (9)$$

The fluid density $\rho_f(T(t, z), P(t))$ can be obtained from an equation of state if temperature and the thermodynamic pressure (assumed $P(t)$ to be constant along z due to the low-Mach number condition) are known. The variation in density may be caused by the fluid accumulation in the system (equivalent to pressure change), which occurs instantaneously along z or by a temperature change.

Analogously, the superficial velocity might be introduced to the model and defined as

$$u = v\phi = \frac{F(t)}{\rho_f(T(t, z), P(t))A} \quad (10)$$

2.4.2. Mass balance for the fluid phase

The detailed derivation of the mass balance equation for the fluid phase can be found in the appendix (A.2). The movement of the mobile pseudo-homogeneous phase (equation 11) is considered only in the axial direction, while the properties of the system in the radial direction are assumed to be uniform. Additionally, the boundary layer adjacent to the inner wall of the extractor is neglected, resulting in a constant velocity profile across any cross-section of the extractor perpendicular to the axial direction. Although the particle size distribution and void fraction of the solid phase may change along the extractor, they are assumed to remain constant in time. Furthermore, the thermodynamic pressure is assumed to be constant along the device due to the Low-Mach number condition, as previously discussed. The amount of solute in the solvent is considered negligible, resulting in the fluid phase being described as pseudo-homogeneous, and its properties are assumed to be the same as the solvent. The mass balance equation for the fluid phase includes convection, diffusion, and kinetic terms.

$$\frac{\partial c_f(t, z)}{\partial t} + \frac{1}{\phi} \frac{\partial (c_f(t, z)u)}{\partial z} = \frac{1-\phi}{\phi} r_e(t, z) + \frac{1}{\phi} \frac{\partial}{\partial z} \left(D_e^M \frac{\partial c_f(t, z)}{\partial z} \right) \quad (11)$$

Here, $c_f(t, z)$, $c_s(t, z)$, and $T(t, z)$ represent the concentration of solute in the fluid phase, the concentration of solute in the solid phase, and temperature, respectively. $r_e(t, z)$ is a mass transfer kinetic term. $F(t)$ is the mass flow rate, $P(t)$ is the pressure, ϵ is the void fraction of the bed, $\rho_f(T(t, z), P(t))$ is the fluid density, ρ_s is the solids density, $D_e^M(T(t, z), P(t), F(t))$ is the axial mass diffusion coefficient, and u is the superficial velocity.

2.4.3. Mass balance for the solid phase

The solid phase is considered to be stationary, with negligible convection and diffusion terms in the mass balance equation (equation 12). Therefore, the only significant term in this equation is the kinetic term (as defined in equation 13), which connects the solid and fluid phases. The extract is represented by a single pseudo-component to simplify the analysis.

$$\frac{\partial c_s(t, z)}{\partial t} = \underbrace{r_e(t, z)}_{\text{Kinetics}} \quad (12)$$

2.4.4. Kinetic term

The kinetic term in this study is based on the two-film theory proposed by Reverchon [1], and the mass transfer kinetic is given by equation 13. This equation takes into account the overall diffusion coefficient and the concentration gradient, which acts as the driving force for the process.

As the solvent flows through the bed, CO_2 molecules diffuse into the pores and adsorb on the particle surface to form an external fluid film around the solid particles due to the solvent-solid matrix interactions. The effect of Knudsen diffusion is negligible in this process, as the mean free path of the molecule is much smaller than the pore diameter. The dissolved solute diffuses from the particle's core through the solid-fluid interface, the pore, and the film into the bulk. Figure 6 illustrates the mass transfer mechanism, where the mean solute concentration in the solid phase is denoted as c_s and the equilibrium concentrations at the solid-fluid interface are denoted as c_s^* and c_p^* , respectively, for solid and fluid phases. The concentration of the solutes in the fluid phase in the center of the pore is denoted as c_p . As the solute diffuses through the pore, its concentration changes and reaches c_{pf} at the opening of the pore. The solute then diffuses through the film around the particle and reaches a concentration in bulk c_f . The two-film theory describes the solid-fluid interface inside the pore. The overall mass transfer coefficient can be determined if the relationship between the solute concentration in one phase and its equilibrium concentration is known.

Bulley et al. [18] suggests a process where the driving force for extraction is given by the difference between the concentration of the solute in bulk, c_f , and in the center of the pore, c_p^* . The concentration c_p^* is in equilibrium with c_s according to an equilibrium relationship. The rate of extraction is thus $r_e(c_f - c_p^*(c_s))$.

On the other hand, Reverchon [1] proposes a driving force given by the difference between c_s and c_p^* . c_p^* is

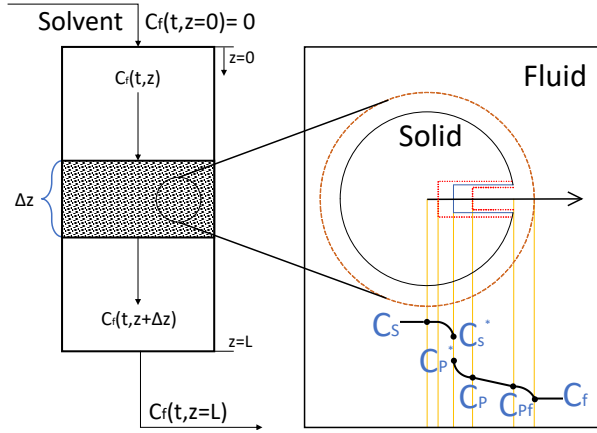


Figure 6: The extraction mechanism

determined by an equilibrium relationship with c_f and the extraction rate is $r_e(c_s - c_p^*(c_f))$ or more precisely

$$r_e(t, z) = \frac{D_i(T(t, z), P(t))}{\mu l^2} (c_s(t, z) - c_p^*(t, z)) \quad (13)$$

where μ is sphericity, l a characteristic dimension of particles and can be defined as $l = r/3$, r is the mean particle radius, ρ_s is the solid density, $D_i(T(t, z))$ corresponds to the overall diffusion coefficient and $c_p^*(t, z)$ is a concentration at the solid-fluid interface (which according to the internal resistance model is supposed to be at equilibrium with the fluid phase).

According to Bulley et al. [18], a linear equilibrium relationship (equation 14) can be used to find an equilibrium concentration of the solute in the fluid phase $c_f^*(t, z)$ is based on the concentration of the solute in the solid phase $c_s(t, z)$

$$c(t, z) = k_p(T(t, z), P(t)) q^*(t, z) \quad (14)$$

The volumetric partition coefficient $k_p(T(t, z), P(t))$ behaves as an equilibrium constant between the solute concentration in one phase and the corresponding equilibrium concentration at the solid-fluid interphase. According to Spiro and Kandiah [19], the term $k_p(T(t, z), P(t))$ can be expressed as the function of mass partition factor $k_m(T(t, z))$.

$$k_m(T(t, z)) = \frac{k_p(T(t, z), P(t)) \rho_s}{\rho(T(t, z), P(t))} \quad (15)$$

Equation 16 represents of the kinetic term according to Reverchon [1]

$$r_e(t, z) = -\frac{D_i(T(t, z), P(t))}{\mu l^2} \left(c_s(t, z) - \frac{\rho_s}{k_m(T(t, z)) \rho_f(T(t, z), P(t))} c_f(t, z) \right) \quad (16)$$

2.4.5. Heat balance

The heat equation was introduced in the previous chapter through equation 5 and in the appendix A.2

$$\begin{aligned} & \frac{\partial (\rho_f(T(t, z), P(t)) e(t, z) A_f)}{\partial t} + \frac{\partial (\rho_f(T(t, z), P(t)) A_f v e(t, z))}{\partial z} \\ &= -P(t) \frac{(A_f v)}{\partial z} + \frac{\partial}{\partial z} \left(\frac{\partial T(t, z)}{\partial z} \right) \end{aligned} \quad (17)$$

Following Elliott [6] or Gmehling et al. [20], a real gas internal energy definition can be obtained from the departure functions, defined through equation 18. In thermodynamics, the departure function is a mathematical function that characterizes the deviation of a thermodynamic property of a real substance from that of an ideal gas at the same temperature and pressure. The departure function is typically defined as the difference between the value of a thermodynamic property for a real fluid and the corresponding value for an ideal gas at the same temperature and pressure. Common departure functions include those for enthalpy, entropy, and internal energy. They are typically computed by integrating a function that depends on the equation of state and its derivatives. More information on the departure functions can be found in appendix A.1.4.

$$de(t, z) = C_v dT - \left[P(t) - T(t, z) \left(\frac{\partial P(t)}{\partial T(t, z)} \right) \right]_{v_m} dv_m \quad (18)$$

where $e^{id}(t, z)$ is the internal energy of perfect gas.

If a gas is considered to be perfectly caloric ($e(t, z) = C_v T(t, z)$), then the energy equation can be written in form of temperature. The perfectly caloric gas can be seen as the special case of a real gas, where the second term of the equation 18 goes to zero.

For real gases, it is complicated to write the heat balance in terms of temperature, but it can be used directly in the form of internal energy, as it is given by equation 5. In such a case, the temperature (which is used as the input to some other functions) needs to be recovered from the internal energy. A relation for the internal energy can be obtained from an equation of state. For Peng-Robinson, such a relation is given by equation 19 as presented by Elliott [6].

$$\frac{e(t, z) - e^{id}(t, z)}{RT(t, z)} = -\frac{A}{B\sqrt{8}} \frac{\kappa\sqrt{T_r}}{\sqrt{\alpha}} \ln \left[\frac{Z + (1 + \sqrt{2})B}{Z + (1 - \sqrt{2})B} \right] \quad (19)$$

To solve equation 19, values of temperature, pressure, and density need to be known. If an equation of state is introduced, then only two out of three variables need to be obtained as the third one can be calculated, this can be represented as follow

$$e(t, z) = e(T(t, z), P(t), \rho_f(T(t, z), P(t))) = e(T(t, z), P(t), \rho(T(t, z), P(t))) \quad (20)$$

If the value of internal energy $e(t, z)$ is known from the time evolution of the energy equation (5), and pressure is known from measurement, then the temperature can be reconstructed. A rootfinder can be used to find a value of temperature, which minimizes the difference between the value of internal energy coming from the time evolution and the output from equation 19. Such a procedure allow to find local temperature along spatial direction z and needs to be repeated every time-step.

Another way to express the energy equation is to introduce enthalpy $h(t, z) = e(t, z) + P(t)/\rho$. By introducing the definition of enthalpy, the energy equation becomes

$$\frac{\partial (\rho_f(T(t, z), P(t)) h(t, z) A_f)}{\partial t} - \frac{\partial (P(t) A_f)}{\partial t}$$

$$+ \frac{\partial (\rho_f(T(t, z), \mathbf{P}(t))h(t, z)A_f v)}{\partial z} - \frac{\partial}{\partial z} \left(k \frac{\partial T(t, z)}{\partial z} \right) \quad (21)$$

The main advantage of this formulation is the presence of term $\partial \mathbf{P}(t)/\partial t$, which allows it to directly affect the system through the change of thermodynamic pressure (which is a control variable). If an equation of state is known, the temperature can be recovered from the enthalpy. The enthalpy is related to the pressure and temperature through the following equation:

$$h(t, z) = h(T(t, z), \mathbf{P}(t), \rho_f(T(t, z), \mathbf{P}(t))) = h(T(t, z), \mathbf{P}(t), \rho(T(t, z), \mathbf{P}(t))) \quad (22)$$

If the value of enthalpy is known from the time evolution and pressure can be measured, then the equation 22 can be solved for the temperature to recover the temperature profile. By applying the departure functions to Peng-Robinson equation of state, the relation 22 can be expressed directly through equation 23 as presented in A.1.4 or given by Gmehling et al. [20].

$$h(t, z) - h(t, z)^{id} = RT(t, z) \left[T_r(Z - 1) - 2.078(1 + \kappa) \sqrt{\alpha} \ln \left(\frac{Z + 2.414B}{Z - 0.414B} \right) \right] \quad (23)$$

The equation 23 requires an reference state, which in this case is assumed to be $T_{ref} = 298.15$ [K] and $P_{ref} = 1.01325$ [bar].

As discussed by Gmehling et al. [20], special attention should be paid to high pressures systems, so the influence of the intermolecular forces on the enthalpy is taken into account. In most cases, these forces are attractive, so additional energy is necessary to move the molecules away from each other, that is, to lower the density. If this energy is not added, the substance cools down when it is expanded.

2.4.6. Extraction yield

The efficiency of the process (the yield) is calculated according to equations 24 as presented by Sovova et al. [21]. The measurement equation evaluate the mass of solute at the outlet of the extraction unit and sums it. The integral form of the measurement equation (24) can be transformed into the differential form (25) and augmented with the process model.

$$y(t) = \int_{t_0}^{t_f} \frac{F(t)}{\rho_f(T(t, z), \mathbf{P}(t))} c_f(t, z) \Big|_{z=L} dt \quad (24)$$

$$\frac{dy}{dt} = \frac{F(t)}{\rho_f(T(t, z), \mathbf{P}(t))} c_f(t, z) \Big|_{z=L} \quad (25)$$

2.4.7. Initial and boundary conditions

It is assumed that the solvent is free of solute at the entrance of the extractor and that all the solid particles have the same initial solute content c_{s0} . The fluid concentration is assumed to not be uniform, and as such it is described by a function $H = H(z)$. The further discussion on the distribution of concentration in the fluid phase can be found in appendix A.5. Moreover, it is considered that the initial temperature of the extractor in every place is the same and described by h_0 . Therefore, the initial conditions employed in the simulation are:

$$\begin{aligned} c_f(t = 0, z) &= H(z) \\ c_s(t = 0, z) &= c_{s0} \\ h(t = 0, z) &= h_0 \end{aligned}$$

2.4.8. State-space representation

The process model can be written in a general form:

$$\begin{bmatrix} \frac{\partial c_f(t, z)}{\partial t} \\ \frac{\partial c_s(t, z)}{\partial t} \\ \frac{\partial h(t, z)}{\partial t} \\ \frac{\partial y(t)}{\partial t} \end{bmatrix} = \begin{bmatrix} \bar{\phi}_1(c_f(t, z), c_s(t, z), h(t, z); \Theta) \\ \bar{\phi}_2(c_f(t, z), c_s(t, z), h(t, z); \Theta) \\ \bar{\phi}_3(c_f(t, z), c_s(t, z), h(t, z); \Theta) \\ \bar{\phi}_4(c_f(t, z), c_s(t, z), h(t, z); \Theta) \end{bmatrix} = \bar{\phi}(t, z; \Theta) = \frac{\partial \chi(t, z)}{\partial t} \quad (27)$$

where Θ is a set of parameters present in the model, $\bar{\phi}$ is a set of functions that correspond to state equations of the model, and χ is the state-space model.

Each function $\bar{\phi}_i$ is transformed to a corresponding set of N_z discretized equations denoted as $\mathbf{G}_{i \times N_z + 1}$ to $\mathbf{G}_{(i+1) \times N_z}$, where i corresponds to the process model equation. The state-space model $\chi(t, z)$ after the discretization is represented by $\dot{\mathbf{x}}(t)$.

$$\dot{\mathbf{x}}(t) = \frac{d\mathbf{x}(t)}{dt} = \begin{bmatrix} \frac{dc_{f,1}(t)}{dt} \\ \vdots \\ \frac{dc_{f,N_z}(t)}{dt} \\ \frac{dc_{s,1}(t)}{dt} \\ \vdots \\ \frac{dc_{s,N_z}(t)}{dt} \\ \frac{dh_1(t)}{dt} \\ \vdots \\ \frac{dh_{N_z}(t)}{dt} \\ \frac{dt(t)}{dt} \end{bmatrix} = \begin{bmatrix} \mathbf{G}_1(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{N_z}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \mathbf{G}_{N_z+1}(c_f(t), c_s(t), T(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{2N_z}(c_f(t), q(t), T(t); \mathbf{p}) \\ \mathbf{G}_{2N_z+1}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{3N_z}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \mathbf{G}_{3N_z+1}(c_f(t), c_s(t), h(t); \mathbf{p}) \end{bmatrix} = \underbrace{\begin{bmatrix} \mathbf{G}_1(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{N_z}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \mathbf{G}_{N_z+1}(c_f(t), c_s(t), T(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{2N_z}(c_f(t), q(t), T(t); \mathbf{p}) \\ \mathbf{G}_{2N_z+1}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \vdots \\ \mathbf{G}_{3N_z}(c_f(t), c_s(t), h(t); \mathbf{p}) \\ \mathbf{G}_{3N_z+1}(c_f(t), c_s(t), h(t); \mathbf{p}) \end{bmatrix}}_{\mathbf{G}(\mathbf{x}(t); \mathbf{p})}$$

where $\mathbf{x} \in \mathbb{R}^{N_x=3N_z}$ and $\mathbf{p} \in \mathbb{R}^{N_p=N_\Theta+N_u}$, N_Θ is the number of model parameters, N_u is the number of control variables.

In a state-space sense, the state variables of the system are the local concentrations of solute in the fluid and solid phases ($c_f(t, z)$ and $c_s(t, z)$, respectively), and the local enthalpy of the pseudo-homogeneous phase ($h(t, z)$). The controllable input variables are the mass flow-rate and temperature of the solvent in the feed ($F_{in}(t) = \mathbf{P}(t)$ and $h_{in}(t) = h(t, z = 0)$, respectively) and the pressure in the extractor ($P(t, z) = \mathbf{P}(t)$). We extend the system state-space by assuming that extraction yield can be modelled as a function of a known initial mass of solute in the solid phase and it can be measured after the separator ($Y(t)$).

The system is controllable by manipulating the flow-rate and temperature (enthalpy) of CO_2 in the feed, and the pressure in the extractor.

2.4.9. Discretization methods

The method of lines is used to transform the process model equations into a set of ODEs denoted as $G(x(t); p)$. The partial derivatives in z-direction are computed using a first-order and second-order finite difference approximation. The backward finite difference is used to approximate the first-order derivative, while the central difference scheme is used to approximate the second-order derivative. The length of the fixed bed is divided into N_z equally distributed points in z-direction.

As presented in Appendix A.2, all the governing can be written in the integral form using the Divergence Theorem. The integral equation states that the change rate of the integral of any quantity over an arbitrary control volume is given by the flux through the boundary of the control volume, with being the outer surface normal through the boundary. That quantity is neither produced nor consumed inside of the control volume and is hence conserved. For a derivative to be conservative, it must form a telescoping series. In other words, after the addition of all terms coming from the discretization over a grid, only the boundary terms should remain and the artificial interior points should cancel out. To ensure the mass conservation, the discretization is applied on the conservative form of the process model.

To ensure the stability of the convergence while solving the partial differential equations, the Courant–Friedrichs–Lewy condition (given by Equation 28) was applied. Based on the discretization step and the velocity, the Courant–Friedrichs–Lewy condition can be used to find.

$$C = \frac{u\Delta t}{\Delta x} \leq C_{max} = 1 \quad (28)$$

where u is velocity, Δt is a time step and Δx is the space discretization.

2.5. Parameter estimation

The goal of parameter estimation is to obtain the "best" estimate of parameter set θ (which is a subset of the parameter space Θ containing all parameters of a model) based on the continuous observations $Y(t)$ or the discrete observations $Y(t_i)$. Conceptually, the unobservable error $\epsilon(t)$ is added to the deterministic model output, $y(t)$ (equation 24), to give the observable dependent variable $Y(t)$ (for example results of an experiments). For discrete observations, this can be expressed as:

$$Y(t_i) = y(\theta, t_i) + \epsilon(t_i)$$

For continuous variables, the equation is:

$$Y(t) = y(\theta, t) + \epsilon(t)$$

However, obtaining analytical solutions for a deterministic process model can be challenging, so experiments are often conducted where the vector of derivatives $dY(t_i)/dt$ is measured instead of $Y(t_i)$ itself. In such cases, it is assumed

that the unobservable error is added to the deterministic derivative $dY(\theta, t_i)/dt$ as shown below

$$\frac{dY(t_i)}{dt} = \frac{dy(\theta, t_i)}{dt} + \epsilon(t_i) \quad (29)$$

In the case where the error in the first observation is denoted as ϵ_1 , the error in the second observation ϵ'_2 incorporates ϵ_1 as well as an independent random component, given by $\epsilon'_2 = \epsilon_1 + \epsilon_2$. Similarly, the error in the third observation is $\epsilon'_3 = \epsilon_1 + \epsilon_2 + \epsilon_3$, and so on. Mandel [22] made a distinction between the typically assumed independent measurement error in the dependent variable and a "cumulative" or interval error, in which each new observation encompasses the error of the previous ones. Cumulative errors arise from fluctuations in the process itself due to small variations in operating conditions and are not independent; only the differences in measurement from one period to the next are independent.

2.5.1. Least squares estimation

In statistics, least squares parameter estimation is a method used to estimate the parameters of a linear regression model. One of the benefits of this method is that it does not require any prior knowledge about the distribution of unobservable errors. Additionally, it yields unbiased estimates and results in the minimum variance among all linear unbiased estimators.

If the model responses, represented by Y , are continuous functions of time from $t = 0$ to $t = t_f$, the Markov criterion, also known as "rigorous least squares," is used to minimize the integral squared error γ . This is achieved by minimizing the following expression:

$$\gamma = \frac{1}{2} \int_0^{t_f} [Y(t) - y(\theta, t)]^T \Gamma^{-1} [Y(t) - y(\theta, t)] dt \quad (30)$$

$$\hat{\theta} = \arg \min_{\theta \in \Theta} \gamma \quad (31)$$

Here, Γ represents the covariance matrix or a matrix of appropriate weights, $\hat{\theta}$ is the parameter estimate, Θ is the parameter space, and γ is the time-integrated value of the error squared.

Alternatively, if the observations are made at discrete time intervals t_i , where $i = 1, 2, \dots, n$, the Markov criterion is to minimize the sum of squared errors, which is given by:

$$\gamma = \frac{1}{2} \sum_{i=1}^n [Y(t_i) - y(\theta, t_i)]^T \Gamma^{-1} [Y(t_i) - y(\theta, t_i)] \quad (32)$$

$$\hat{\theta} = \arg \min_{\theta \in \Theta} \gamma \quad (33)$$

The parameter estimate, $\hat{\theta}$, is obtained by minimizing γ over the parameter space Θ .

If the covariance matrix, Γ , is a diagonal matrix, the Markov criterion becomes a "weighted least squares" criterion. If $\Gamma = \sigma^2 \mathbf{I}$, where σ is a standard deviation and \mathbf{I} is the identity matrix, the Markov criterion becomes the "ordinary least squares" criterion.

2.5.2. Maximum likelihood estimation

Maximum likelihood estimation (MLE) is a statistical method used to estimate the parameters of a probability distribution based on observed data. The MLE works by

finding the values of the parameters that maximize the likelihood function, which is the probability of observing the given data for a given set of parameter values. The MLE has desirable properties such as asymptotic efficiency and normality. Although the MLE has often been associated with the normal distribution for mathematical convenience, it can be applied to a wide range of probability distributions.

To find the maximum likelihood estimates, we maximize the joint probability density function, or likelihood function, denoted as $p(\theta|y(t_1), y(t_2), \dots, y(t_n))$, where θ represents the parameters and $y(t_1), y(t_2), \dots, y(t_n)$ represent the observed data. The conditions at the maximum can be refined by incorporating prior information. The posterior probability density function $p(\theta|y)$ can be expressed as the ratio of two probability densities (113) using the continuous variable analogue of Bayes' theorem (details in appendix A.3). In such a case the posteriori distribution is given by Equation 34.

$$p(\theta|y(t_n), \dots, y(t_1)) = \frac{p(\theta, y(t_n), \dots, y(t_1))}{p(y(t_n), \dots, y(t_1))} \quad (34)$$

The numerator of the right-hand side of Equation 34 using Equation 114a becomes

$$p(\theta, y(t_n), \dots, y(t_1)) = p(y(t_n)|\theta, y(t_{n-1}), \dots, y(t_1)) \cdot p(\theta, y(t_{n-1}), \dots, y(t_1)) \quad (35)$$

These operations can be continued repetitively until we get

$$p(\theta, y(t_n), \dots, y(t_1)) = p(\theta) \prod_{i=1}^n p(y(t_i)|\theta, y(t_{i-1}), \dots, y(t_1)) \quad (36)$$

Examination of Equation 29 shows that $dY(t_i)/dt$ depends only on t_i , θ and $\epsilon(t_i)$ and is not conditioned by any previous measurement. Consequently, we can write

$$p(y(t_i)|\theta, y(t_{i-1}), \dots, y(t_1)) = p(y(t_i)|\theta) \quad (37)$$

provided Equation 29 is observed as a constraint. The desired joint conditional probability function is thus

$$p(\theta|y(t_n), \dots, y(t_1)) = \frac{p(\theta) \prod_{i=1}^n p(y(t_i)|\theta)}{p(y(t_n), \dots, y(t_1))} \quad (38)$$

We can get rid of the evidence term $p(y(t_n), \dots, y(t_1))$ because it's constant with respect to the maximization. Moreover, if we are lacking a prior distribution over the quantity we want to estimate, then $p(\theta)$ can be omitted. In such a case:

$$p(\theta|y(t_n), \dots, y(t_1)) = \prod_{i=1}^n p(y(t_i)|\theta) = \prod_{i=1}^n L(\theta|y(t_i)) \quad (39)$$

The likelihood function $L(\theta|y)$ for the parameters based on several observations is the product of the individual functions if the observations are independent.

$$L(\theta|y(t_n), \dots, y(t_1)) = \prod_{i=1}^n L(\theta|y(t_i)) \\ = p(y(t_1)|\theta) p(y(t_2)|\theta) \dots p(y(t_n)|\theta) \quad (40)$$

In choosing as estimates of θ the values that maximize L for the given values $(y(t_i))$, it turns out that it is more convenient to work with the $\ln L$ than with L itself:

$$\ln L = \ln p(y(t_1)|\theta) + \ln p(y(t_2)|\theta) + \dots + \ln p(y(t_n)|\theta) = \sum_{i=1}^n \ln p(y(t_i); \theta) \quad (41)$$

By assuming that the conditional distribution of \bar{Y}_i , given y_i , is normal, then we form the likelihood function based on the probability density:

$$p(\theta, \sigma|y(t_n), \dots, y(t_1)) = \prod_{i=1}^n \frac{1}{\sqrt{2\pi}\sigma} \exp \left[-\frac{1}{2\sigma^2} (Y(t_i) - y(\theta, t_i))^2 \right] \\ L(\theta, \sigma|y(t_n), \dots, y(t_1)) = \prod_{i=1}^n \frac{1}{\sqrt{2\pi}\sigma} \exp \left[-\frac{1}{2\sigma^2} (Y(t_i) - y(\theta, t_i))^2 \right] \quad (42)$$

where σ is the variance

By taking the natural logarithm of the Equation 42, the final form of the objective function can be obtained:

$$\ln L = -\frac{n}{2} (\ln \sqrt{2\pi} + \ln \sigma^2) - \frac{\sum_{i=1}^n [Y(t_i) - y(\theta, t_i)]^2}{2\sigma^2} \quad (43)$$

The parameter estimation problem can be formulated as follow:

$$\begin{aligned} \hat{\theta}_{MLE} &= \arg \max_{\sigma, \theta \in \Theta} \ln L = \arg \max_{\sigma, \theta \in \Theta} p(\theta|y) \\ \text{subject to} & \quad \dot{x} = f(t, x, \theta) \\ & \quad \dot{\theta} = 0 \\ & \quad y = y(x) \\ & \quad \theta^{lb} \leq \theta \leq \theta^{ub} \end{aligned} \quad (44)$$

where $\hat{\theta}$ is as maximum likelihood estimator, θ^{lb} define the minimal value of θ and θ^{ub} is the maximum value of θ .

Based on the first order optimality condition, the $\ln L$ can be maximized with respect to the vector θ by equating to zero the partial derivatives of $\ln L$ with respect to each of the parameters:

$$\frac{\partial \ln L}{\partial \theta} = \frac{\partial \sum_{i=1}^n \ln p(y(t_i)|\theta)}{\partial \theta} = 0 \quad (45)$$

Solution of Equations 45 yield the desired estimates $\hat{\theta}$. For some models, these equations can be explicitly solved for $\hat{\theta}$ but in general no closed-form solution to the maximization problem is known or available, and an maximum likelihood estimator can only be found via numerical optimization.

2.6. Experimental work

In order to solve the optimization problem presented in equation 44, it is necessary to have knowledge of the dataset $Y(t)$, which was obtained by extracting caraway oil from caraway seeds. To prepare the seeds for extraction, they were first pre-treated using a Retsch SM 300 cutting mill to reduce their particle size to 1mm and break their outer shell. The moisture content of the seeds was then determined using an Infrared Moisture Analyser, which revealed an average moisture content of 4.83% in the solid particles after grinding. Next, the density of the solid material was measured using a pycnometer, which can be found in appendix A.6. Finally, the material's porosity was also calculated to be 0.5, as explained in appendix A.7.

The caraway seeds used in the experiments were obtained from Caraway Finland during the 2022 harvesting

Time min	40°C/200 bar	50°C/200 bar	40°C/300 bar	50°C/300 bar
	Mass g	Mass g	Mass g	Mass g
0	0.0	0.0	0.0	0.0
5	1.7	1.8	1.1	1.2
10	5.7	4.7	4.7	4.5
15	10.9	9.5	10.8	11.3
20	17.8	15.1	20.6	20.6
25	24.9	21.3	32.1	30.6
30	30.3	26.4	40.5	37.8
35	34.0	29.8	46.7	43.3
40	37.5	33.1	50.5	47.5
45	40.1	35.6	53.1	50.8
50	43.3	38.7	55.5	53.7
55	45.5	41.5	57.6	56.5
60	47.6	43.5	59.3	58.1
65	49.2	45.9	60.9	59.6
70	50.8	48.3	62.2	61.1
75	52.6	50.8	63.7	62.7
80	54.2	52.9	64.9	64.4
85	55.1	55.3	66.6	65.8
90	56.2	57.2	67.6	66.7
95	57.3	58.3	68.9	67.4
100	58.3	59.4	69.7	68.2
105	59.4	60.4	70.6	69.1
110	60.2	61.6	71.2	70.1
115	61.4	62.7	71.9	70.9
120	62.5	63.9	72.3	71.6
125	63.3	64.7	72.8	72.3
130	64.0	65.4	73.2	72.9
135	64.8	65.9	73.6	73.5
140	65.5	66.4	74	74
145	66.2	67.1	74.3	74.5
150	66.8	67.7	74.6	74.9

Table 1
Yield data

season. The experiments were conducted using a 10-litre extractor with an inner diameter of approximately 15 cm and a height of 60 cm, which was rented from VTT Bioruukki. Four experiments were performed under different operating conditions: 40°C/200 bar, 50°C/200 bar, 40°C/300 bar, and 50°C/300 bar. The volumetric flow rate used in all experiments was on average 0.4 litres per minute, with negligible variations (up to 5%). The amount of solid material used for extraction was 1 kg (or 1.6 litres), which was not enough to fill the entire extraction chamber.

After loading the material into the chamber, the extractor was pre-heated to the desired temperature. The outlet line was then closed, and the extraction chamber was filled with CO_2 . The CO_2 was pumped and compressed until the desired operating pressure was reached. When the operating temperature and pressure were achieved, the outlet line was opened, allowing the solvent to flow through the system. The solvent extracted the essential oils from the solid material, and the resulting CO_2 and oil mixture flowed from the extractor to the separator. The separator operated at 50°C and 50 bar, with gaseous CO_2 exiting from the top and oil remaining at the bottom. The extraction time for each batch was 150 minutes, measured from the opening of the extractor outlet port. Every 5 minutes, the oil was drained from the separator, and its weight was measured. The resulting time series $Y(t)$ corresponds to the output of measurement equation 24 and can be used for parameter estimation.

The results of experiments are presented in the Table 1.

3. Results

The parameter estimation problem was solved by fitting the process model to the dataset given in the Table 1. Each time-series was fitted to the model separately. As a result of fitting, the following parameters were obtained:

- Partition coefficient: k_m
- Internal Diffusion coefficient: D_i
- Axial Diffusion coefficient: D_e^M
- Standard Deviation: σ

Moreover, the initial state estimation was performed together with parameter estimation. The concentration of solute in the solid phase is assumed to be constant and uniformly distributed. On the other hand, the solute concentration in the fluid phase should not follow the same assumption as the solid phase. During the preparation period the solute diffuses to the fluid phase in contact with the solid particles. Later, the solute in the fluid phase is partially moved (if the pressure increase in the system, the pump cause the movement of the fluid, even if the outlet valve is closed) along the extractor. As a result, the distribution of solutes mass in the fluid phase is assumed to not be uniform. Some conclusions can be drawn from the analysis of the initial part of each yield curve. It can be noticed that each curve at the beginning has a curvature, which is not linear. In a general sense, it can be said that a quadratic function could approximate the initial part of each extraction curve. A function that, after integration, gives a quadratic-like result is a straight line. Based on that observation, the solute concentration in the fluid phase is assumed to be linearly distributed. The solute concentration is assumed to be zero at the outlet and reach the maximum at the beginning of the fixed bed. The details on the calculation are given in appendix A.5. The linear distribution can be defined if the total mass of solute m_{total} and partition coefficient τ are known.

Due to initial state estimation, two additional parameters are fitted.

- Total mass of solute: m_{total}
- Partition coefficient: τ

To ensure that parameters found by the optimizer do not reach unrealistic values, an additional set of inequality constraints is introduced. The lower and upper bounds for each parameter are given in Table 2. The same Table contains the initial guesses for the optimizer. To ensure that the solution is a global solution multiple initial points have been tested.

The figure 7 show yield curves for each of experiments. The blue dots indicate the data point obtained as a measurement from the laboratory experiments. The black curve represents the yield curve obtained from the initial guess of the parameters. The blue curve correspond to yield curve obtained as a solution of the optimization problem.

Parameter	k_m	D_i	D_e^M	m_{total}	τ	σ
Lower bound	0	0	0	76	0	0
Upper bound	1	$+\infty$	$+\infty$	100	1	$+\infty$
Initial guess	1	3	1	77	0.65	0.1

Table 2

Constraints of the parameter estimation problem

Parameter	40[C]200[bar]	50[C]200[bar]	40[C]300[bar]	50[C]300[bar]
k_m [-]	1	0.99999	1	1
D_i [m^2/s]	3.8288	3.54028	8.80791	7.07805
D_e^M [m^2/s]	6.66651	7.37702	1.41263	3.079
m_{total}	76	76.00001	76	76
τ	0.69145	0.74664	0.64535	0.64394
σ	0.32387	0.34538	0.5503	0.37304

Table 3

Parameter estimation results

The curves on the left hand-side of the Figure 7, are the cumulative yield curves, which represents the cumulative amount of the oil collected during each experiment. The Figures on the right hand-side, represent the derivative of the cumulative yield curves. As explained in the chapter 2.5, the derivatives of the cumulative measurements are independent on the previous measurement and should be used for the parameter estimation.

The results of the parameter estimation can be found in Table 3. As can be noticed, the optimizer found that the partition factor should be as high as possible and moved values of k_m for each experiment to unity, which was chosen as the upper bound.

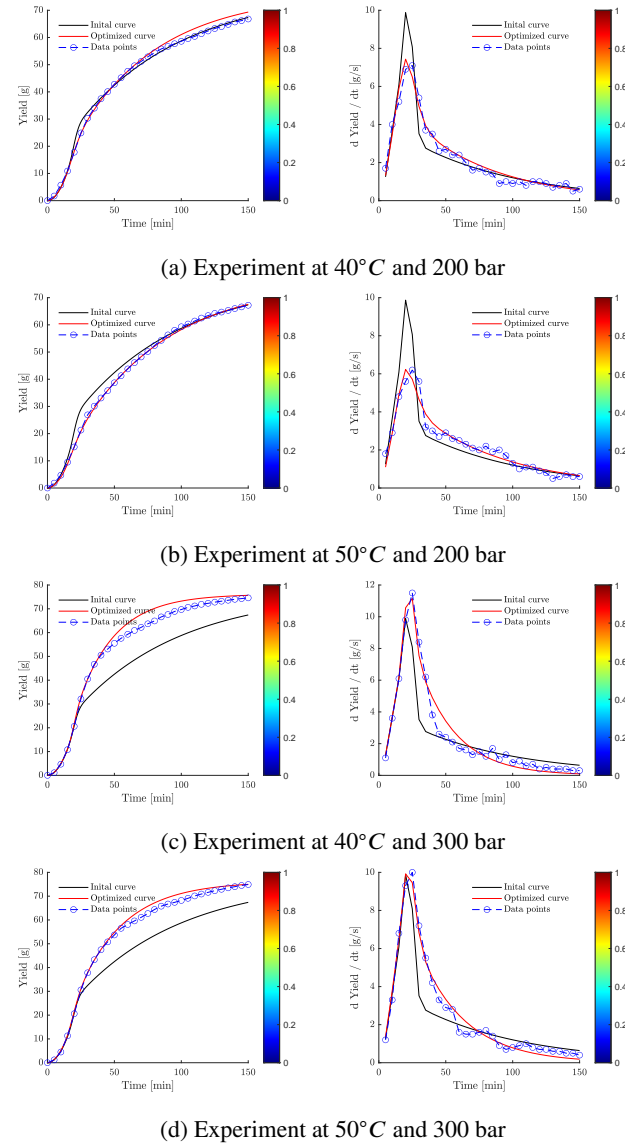
Values of the internal diffusion coefficients are distinguished for each experiment, with an increasing trend with respect to the fluid density. The order of magnitude of D_i obtained from the optimization is similar to values found by other researchers. Reverchon [1] performed the parameter estimation for the extraction process of sage oil from seeds and reported $D_i \approx 6e - 13$.

The axial diffusion coefficient obtained from the optimizer seems to be quite high if compared to the internal diffusion coefficients. Nevertheless, the values of D_e^M have similar order of magnitude as reported in literature, for example Reis-Vasco et al. [23]. The relatively high values of the axial diffusion coefficients might be justified by taking into account low values of flow rate used in all the experiments.

As the total amount of the oil present in the system is unknown, and as such it has been obtained from the optimizer. Given the dataset and the process model, the optimizer found that all most all the oil was extracted out of the seeds and that the best fit is obtained if the total mass of the oil reach a lower bound equal to 76 g.

As explained above, and in Appendix A.5, the linear distribution of can be fully determined if the parameter τ is known. The split ratio for experiments conducted at 40°C / 300 bar and 50°C / 300 bar are similar to each other and close to 0.64. The experiments performed at 40°C / 200 bar and 50°C / 200 bar have τ equals to 0.69 and 0.74, respectively.

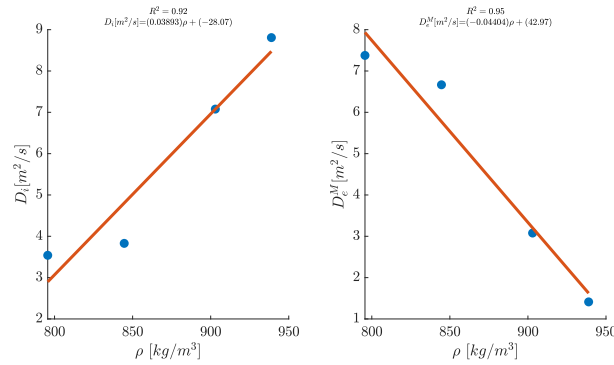
The noise present in each dataset is quantified by parameter σ . It can be observed dataset obtained at 40°C / 200 bar, 50°C / 200 bar and 40°C / 300 bar have similar value of $\sigma \approx 0.34$. The visual investigation of the dataset obtained at 50°C / 300 bar allow to explain higher value of σ for this dataset. This dataset have higher deviation of the data points from the simulated curve.


Figure 7: Results of parameter fitting, with estimation of the initial state

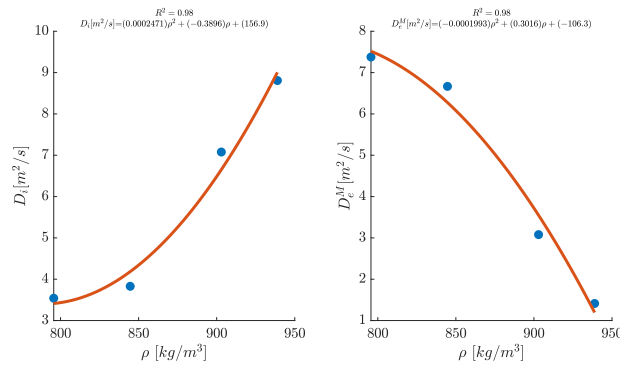
The results obtained from the parameter estimation are used to develop a generalized process model. This model expresses the parameters D_i and D_e^M as functions of the operating conditions, such as temperature (T), pressure (P), and flow rate (F). Since the experiments were performed under varying temperature and pressure, the influence of these variables is taken into account. To simplify the problem, the correlations are written as functions of fluid density rather than temperature and pressure.

While the experimental procedure and device can affect τ and σ , these parameters are not further considered. They are used only to define the initial conditions and are not included in the process model. Additionally, since the optimizer found that k_m and m_{total} have the same value, they are assumed to be constants and not considered further.

The other two parameters (D_i and D_e^M) are subject to regression to obtain correlations in terms of density. Figure 8 shows the first- and second-order polynomials used to fit D_i and D_e^M as a function of fluid density. Interestingly, these two parameters exhibit opposite behavior.



(a) First order polynomial regression of fitted parameters as a function of fluid density ρ_f



(b) Second order polynomial regression of fitted parameters as a function of fluid density ρ_f

Figure 8: Results of parameter fitting, with estimation of the initial state

The results of the linear regression applied to D_i and D_e^M are presented in Figure 8a. It can be observed that the internal diffusion coefficient increases with increasing fluid density, while the axial diffusion coefficient decreases.

The increase in the density of the supercritical fluid can lead to an increase in the solubility of the solute and, consequently, an increase in the concentration gradient. This increased concentration gradient drives the solute to diffuse more rapidly through the matrix of the solid particles, resulting in an increase in the internal mass transfer coefficient. However, the effect of increased solubility on the internal

mass transfer coefficient is greater than the resistance coming from higher density, resulting in an overall increase in the internal mass transfer coefficient with density.

The behaviour of the axial diffusion coefficient can be explained through Graham's Law of Diffusion, which states that the rate of diffusion of a molecule is inversely proportional to the square root of its density under given conditions of temperature and pressure. This means that denser media slow down the axial diffusion rate of molecules.

IDEAL I SHOULD SUPPORT THIS EXPLANATION WITH SOME REFERENCES

Figure 8a show the result of fitting a second-order polynomial in density space. It can be observed that the fit obtained from quadratic functions seems to be represent both dataset better.

4. Conclusions

In this work, a dataset obtained from experiments conducted at various temperature and pressure was utilized for parameter estimation to develop a process model. The model showed good agreement with experimental yield curves for the extraction of caraway oil from caraway seeds. Linear and non-linear regression techniques were employed to establish correlations for the internal diffusion coefficient and axial diffusion coefficient in terms of fluid density. These correlations allowed for the generalization of the process model to a range of operating conditions, specifically temperature ranging from 40°C to 50°C , pressure ranging from 200 bar to 300 bar, and flow rate of 0.4 l/min (with flow variations up to 5%). The generalized model can be used to evaluate the impact of different operating conditions on the extraction yield or for economic analysis of the extraction process.

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A. Appendix

A.1. Thermodynamic

A.1.1. Equation of state and properties of the fluid phase

We consider equations of state in the general form $P(t)v_m[T(t, z), P(t)] = ZRT(t, z)$, where v_m denotes the molar volume of CO_2 , Z represents its compressibility factor, and R is the universal gas constant. More specifically, we are interested in these equations because of the possibility to express the compressibility Z as an explicit function of temperature and pressure. This is the case when Z is obtained as one of the physically meaningful roots of a polynomial equation, like the Van der Waals's, Peng-Robinson's or Redlich-Kwong's equation of state. In this work the Peng-Robinson equation of state (Peng and Robinson [5]) is used

In Peng-Robinson's equation of state, the compressibility $\bar{Z}[T(t, z), P(t)]$ solves the third-order polynomial equation given by equation 46. The polynomial equation can be solved by numerical methods like Newton-Raphson, or by applying Cardano formula. The Cardano formula is preferable in comparison to numerical root-finders, as it allows to obtain an analytical solution. The details of the Cardano formula are presented in appendix A.4.

$$Z^3 - [1 - B[T(t, z), P(t)]] Z^2 + [A[T(t, z), P(t)] - 2B[T(t, z), P(t)] - 3B[T(t, z), P(t)]^2] Z = 0 \quad (46)$$

where $A[T(t, z), P(t)]$ and $B[T(t, z), P(t)]$ are functions of time and space defined on the attraction parameter, $a[T(t, z)] = a_{\text{CO}_2}^c \alpha[T(t, z)]$ with $a_{\text{CO}_2}^c \approx 0.45724 R^2 T_{\text{CO}_2}^{c2} / P_{\text{CO}_2}^c$ and the repulsion parameter, $b_{\text{CO}_2} \approx 0.07780 R T_{\text{CO}_2}^c / P_{\text{CO}_2}^c$, both functions of the critical temperature $T_{\text{CO}_2}^c$ and pressure $P_{\text{CO}_2}^c$. Specifically, we have

$$A[T(t, z), P(t)] = \frac{\alpha[T(t, z)] a_{\text{CO}_2}^c P(t)}{R^2 T^2(t, z)}; \quad (47a)$$

$$B[T(t, z), P(t)] = \frac{b_{\text{CO}_2} P(t)}{RT(t, z)}. \quad (47b)$$

The quantity $\alpha[T(t, z)] = \left[1 + \kappa_{\text{CO}_2} \left[1 - \sqrt{T(t, z)/T_{\text{CO}_2}^c}\right]\right]^2$, with constant $\kappa_{\text{CO}_2} = 0.37464 + 1.54226 \omega_{\text{CO}_2} - 0.26992 \omega_{\text{CO}_2}^2$, is a dimensionless correction term defined on the acentric factor $\omega_{\text{CO}_2} = 0.239$ of CO_2 molecules.

By denoting the physical constants as $\varphi_Z = (R, T_{\text{CO}_2}^c, P_{\text{CO}_2}^c, \kappa_{\text{CO}_2})$, we obtain a spatio-temporal representation of the compressibility $\bar{Z}[T(t, z), P(t) | \varphi_Z]$ with its complete set of functional dependencies and parameters.

A.1.2. Density of the fluid phase

The density ρ_F of the fluid phase is assumed to be equal to the density of solvent, at given temperature and pressure. Because temperature $T(t, z)$ of the fluid phase is a modelled variable, we allow for the density to vary along the bed and in time. From an equation of state of the form $P(t)v_m[T(t, z), P(t)] = ZRT(t, z)$, we get

$$\rho_F[T(t, z), P(t) | \varphi_{\rho_F}] = \frac{P(t)M_{\text{CO}_2}}{RT(t, z)\bar{Z}[T(t, z), P(t) | \varphi_Z]}, \quad (48)$$

where M_{CO_2} denotes the molar mass of CO_2 and $\bar{Z}[T(t, z), P(t)]$ is the compressibility factor that solves Eq. (46). The density of the fluid is thus a function of space and time, due to its dependence on temperature and pressure, and it is expressed in terms of the set of physical constants $\varphi_{\rho_F} = (R, M_{\text{CO}_2}, T_{\text{CO}_2}^c, P_{\text{CO}_2}^c, \omega_{\text{CO}_2})$.

A.1.3. Heat capacity of the fluid phase

The specific heat C_p^F can be calculated from the equation of state, again under the assumption that the fluid phase consists of pure carbon dioxide and that the specific heat of real fluids can be calculated from an ideal contribution plus a residual term Pratt [7]. In the following, we report only the main steps in the derivation of $C_p^{\text{CO}_2}[T(t, z), P(t)]$:

$$C_v^{\text{CO}_2}[T(t, z), P(t)] = C_v^I[T(t, z), P(t)] + C_v^R[T(t, z), P(t)]; \quad (49a)$$

$$C_p^{\text{CO}_2}[T(t, z), P(t)] = \underbrace{C_p^I[T(t, z), P(t)]}_{\text{Eq. (50)}} + \underbrace{C_p^R[T(t, z), P(t)]}_{\text{Eq. (51)}}. \quad (49b)$$

$C_v^{\text{CO}_2}[T(t, z), P(t)]$ and $C_p^{\text{CO}_2}[T(t, z), P(t)]$ are the specific heat of CO_2 at constant volume and pressure, respectively. $C_v^I[T(t, z), P(t)]$ and $C_p^I[T(t, z), P(t)]$, with $C_p^I(T(t, z)) - C_v^I(T(t, z)) = R$, are the specific heat of an ideal gas at constant volume and pressure. $C_v^R[T(t, z), P(t)]$ and $C_p^R(T(t, z), P(t))$ are the correction terms.

For CO_2 (), we have the ideal gas contribution to the specific heat at constant $P(t)$, as function of $T(t, z)$,

$$C_p^I[T(t, z), P(t)] = C_{P0} + C_{P1}T(t, z) + C_{P2}T^2(t, z) + C_{P3}T^3(t, z) \quad (50)$$

where the coefficients of the expansion are $C_{P0} = 4.728$, $C_{P1} = 1.75 \times 10^{-3}$, $C_{P2} = -1.34 \times 10^{-5}$, and $C_{P3} = 4.10 \times 10^{-9}$. For the correction term $C_p^R[T(t, z), P(t)]$ at constant pressure $P(t)$, we have

$$C_p^R[T(t, z), P(t)] = \underbrace{C_v^R[T(t, z), P(t)]}_{\text{Eq. (55)}} + \underbrace{T(t, z) \left(\frac{\partial P(t)}{\partial T} \right)_{v_m(t, z)}}_{\text{Eq. (54)}} \underbrace{\left(\frac{\partial v_m[T(t, z), P(t)]}{\partial T} \right)_{P(t)}}_{\text{Eq. (52)}} - R. \quad (51)$$

The braced terms are obtained from the chosen equation of state $P(t)V[T(t, z), P(t)] = Z[T(t, z), P(t)]RT(t, z)$.

For the partial derivative of the volume with respect to temperature T at constant pressure $P(t)$, we have

$$\left(\frac{\partial v_m[T(t, z), P(t)]}{\partial T} \right)_{P(t)} = \frac{Z[T(t, z), P(t)] R}{P(t)}$$

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$$+ \underbrace{\frac{RT(t, z)}{P(t)} \left(\frac{\partial Z [T(t, z), P(t)]}{\partial T} \right)}_{\text{Eq. (53)}} \bigg|_{P(t)} \quad (52)$$

with partial derivative of the compressibility factor with respect to temperature T at constant pressure $P(t)$

$$\left(\frac{\partial Z [T(t, z), P(t)]}{\partial T} \right)_{P(t)} = \left(\frac{\partial \frac{P(t)v_m [T(t, z), P(t)]}{RT(t, z)}}{\partial T} \right)_{P(t)} \quad (53)$$

Similarly, for the partial derivative of the pressure with respect to temperature at constant volume, we have

$$\left(\frac{\partial P(t)}{\partial T} \right)_{v_m(t, z)} = \left(\frac{\partial \frac{ZRT(t, z)}{v_m [T(t, z), P(t)]}}{\partial T} \right)_{v_m(t, z)} \quad (54)$$

The residual specific heat at constant volume is obtained, by definition, by using the residual internal energy

$$C_v^R [T(t, z), P(t)] = \left(\frac{\partial U^R [T(t, z), P(t)]}{\partial T} \right)_{v_m(t, z)} \quad (55)$$

By denoting the physical constants as $\varphi_{C_p^F} = (R, M_{\text{CO}_2}, T_{\text{CO}_2}^c, P_{\text{CO}_2}^c, \omega_{\text{CO}_2})$, we get the spatio-temporal representation $C_p^F [T(t, z), P(t) | \varphi_{C_p^F}]$ of the specific heat of the fluid phase.

A.1.4. Departure functions for enthalpy calculations

In thermodynamics, a departure function is a concept used to calculate the difference between a real fluid's thermodynamic properties and those of an ideal gas, given a specific temperature and pressure. Common departure functions include those for enthalpy, entropy, and internal energy. These functions are used to calculate extensive properties, which are properties computed as a difference between two states.

For example, to evaluate the enthalpy change between two points, $h(V_1, T_1)$ and $h(V_2, T_2)$, we first calculate the enthalpy departure function between the initial volume V_1 and infinite volume at temperature T_1 . We then add to that the ideal gas enthalpy change due to the temperature change from T_1 to T_2 , and finally subtract the departure function value between the final volume V_2 and infinite volume.

Departure functions are computed by integrating a function that depends on an equation of state and its derivative. The general form of the enthalpy equation is given by:

$$\frac{h^{id} - h}{RT} = \int_{v_m}^{\infty} \left[T \left(\frac{\partial Z}{\partial T} \right)_{v_m} \right] \frac{dv_m}{v_m} + 1 - Z \quad (56)$$

Here, h^{id} represents the enthalpy of an ideal gas, h is the enthalpy of a real fluid, R is the universal gas constant, T is temperature, v_m is the molar volume, and Z is the compressibility factor.

The integral in the equation is evaluated over the range of molar volumes from v_m to infinity. The integral includes a term that depends on the derivative of the compressibility factor with respect to temperature, evaluated at the molar

volume v_m . Finally, the term $1 - Z$ is added to account for the deviation of the fluid's properties from those of an ideal gas.

The Peng–Robinson equation of state relates the three interdependent state properties pressure P , temperature T , and molar volume v_m . From the state properties (P, v_m, T) , one may compute the departure function for enthalpy per mole (denoted h) as presented by Gmehling et al. [20] or Elliott [6]:

$$h - h^{id} = RT \left[T_r(Z - 1) - 2.078(1 + \kappa) \sqrt{\alpha} \ln \left(\frac{Z + 2.414B}{Z - 0.414B} \right) \right] \quad (57)$$

A.2. Governing equations

A.2.1. Mass continuity

Following the work of Anderson [15], the governing equations for compressible fluid with non-uniform cross-section can be obtained. Let's assume that any properties of the flow are uniform across any given cross-section of an extractor. The variation of the cross-section might result from the partial filling of an extractor or its irregular shape. In reality, such a flow is two-dimensional because the area changes as a function of z , and there is a flow-field variation in both directions. The assumption of quasi-one-dimensional flow dictates that the flow properties are a function of z only. The equations described by quasi-one-dimensional assumption hold: (1) mass conservation, (2) Newton's second law, and (3) energy conservation. The modified governing equation can be derived to ensure that these physical principles are satisfied. Let's start with the integral form of the continuity equation:

$$\frac{\partial}{\partial t} \iiint_{V_f} \rho_f dV_f + \iint_S \rho_f \mathbf{V} \cdot d\mathbf{S} = 0 \quad (58)$$

We apply this equation to the shaded control volume shown in Fig. 9.

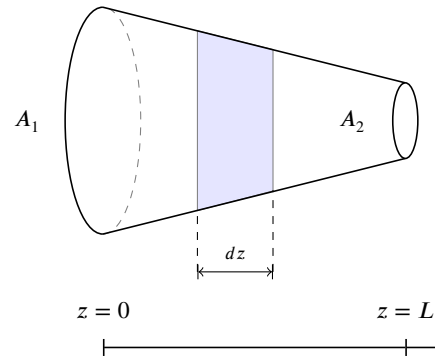


Figure 9: Control volume for deriving the partial differential equation for unsteady, quasi-one-dimensional flow

This control volume is a slice of an extractor, where the infinitesimal thickness of the slice is dz . On the left side of the control volume, consistent with the quasi-one-dimensional assumptions, the density, velocity, pressure, and internal energy denoted by ρ_f , v , P , and e , respectively, are uniform over the area A . Similarly, on the right side of the control volume, the density, velocity, pressure, and internal energy $\rho_f + d\rho_f$, $v + dv$, $P + dP$, and $e + de$, respectively,

are uniform over the area available for fluid phase $A_f + dA_f$. Applied to the control volume in Fig. 9, the volume integral in Eq. 58 becomes, in the limit as dz becomes very small,

$$\frac{\partial}{\partial t} \iiint_{V_f} \rho_f dV_f = \frac{\partial}{\partial t} (\rho_f A_f dz) \quad (59)$$

where $A dz$ is the volume of the control volume in the limit of dz becoming vanishingly small. The surface integral in Eq. 58 becomes

$$\iint_S \rho_f \mathbf{V} \cdot d\mathbf{S} = -\rho_f v A_f + (\rho_f + d\rho_f)(v + dv)(A_f + dA_f) \quad (60)$$

The minus sign on the leading term on the right-hand side is due to the vectors \mathbf{V} and $d\mathbf{S}$ pointing in opposite directions over the left of the control volume, and hence the dot product is negative. Expanding the triple product term

$$\begin{aligned} \iint_S \rho_f \mathbf{V} \cdot d\mathbf{S} = & -\rho_f v A_f + \rho_f v A_f + \rho_f v dA_f + \rho_f A_f dv \\ & + \rho_f dv dA_f + v A_f d\rho_f + v d\rho_f dA_f + A_f d\rho_f dv + d\rho_f dv dA_f \end{aligned} \quad (61)$$

In the limit as dz becomes very small, the terms involving products of the differential in Eq. 61, such as $\rho_f dv dA_f$, $d\rho_f dv dA_f$, go to zero much faster than those terms involving only one differential. Hence, all terms involving products of differentials can be dropped, yielding in the limit as dz becomes very small

$$\iint_S \rho_f \mathbf{V} \cdot d\mathbf{S} = \rho_f v dA_f + \rho_f A_f dv + v A_f d\rho_f \quad (62)$$

Substituting Eqs. 59 and 62 into 58, we have

$$\frac{\partial (\rho_f A_f)}{\partial t} + \frac{\partial (\rho_f A_f v)}{\partial z} = 0 \quad (63)$$

The above partial differential equation form of the continuity equation is suitable for unsteady, quasi-one-dimensional flow. It ensures that mass is conserved for this mode of flow. The $A_f(z)$ is an arbitrary function that describes a change in the cross-section of an extractor. The function $A_f(z)$ can be defined as $A_f(z) = \mathbf{A}\phi(z)$, where ϕ is the bed porosity and \mathbf{A} is the cross-section of an empty extractor.

$$\frac{\partial (\rho_f \mathbf{A}\phi(z))}{\partial t} + \frac{\partial (\rho_f \mathbf{A}\phi(z)v)}{\partial z} = 0 \quad (64)$$

The equation can be simplified by canceling out a constant \mathbf{A}

$$\frac{\partial (\rho_f \phi(z))}{\partial t} + \frac{\partial (\rho_f \phi(z)v)}{\partial z} = 0 \quad (65)$$

If so-called superficial velocity is defined as $u = \phi v$, the mass continuity becomes

$$\frac{\partial (\rho_f \phi(z))}{\partial t} + \frac{\partial (\rho_f u)}{\partial z} = 0 \quad (66)$$

A.2.2. Transport of a species

The transport of a chemical species, in this case, a solute, can be described by an analogous equation to the Eq. 58 with additional terms on the right-hand side. The first term on the right-hand side describes that a substance goes from high-density regions to low-density regions and

is based on the Fick's law ($J_{diff} = D_e^M \frac{\partial c_f}{\partial z}$). The other term corresponds to the mass transfer between solid and fluid phases, which is treated as a source term.

$$\frac{\partial}{\partial t} \iiint_{V_f} c_f dV_f + \iint_S c_f \mathbf{V} \cdot d\mathbf{S} = \iint_S J_{diff} \cdot \mathbf{n} d\mathbf{S} + \frac{\partial}{\partial t} \iiint_{V_s} c_s dV_s \quad (67)$$

Similarly to the continuity equation, in the limit as dz becomes very small

$$\frac{\partial}{\partial t} \iiint_{V_f} c_f dV_f = \frac{\partial}{\partial t} (c_f A_f dz) \quad (68)$$

$$\frac{\partial}{\partial t} \iiint_{V_s} c_s dV_s = \frac{\partial}{\partial t} (c_s A_s dz) \quad (69)$$

The surface integrals in the limit of dz become

$$\iint_S c_f \mathbf{V} \cdot d\mathbf{S} = c_f v dA_f + c_f A_f dv + v A_f dc_f \quad (70)$$

From the Divergence theorem in multi-variable calculus, we have

$$\iint_S J_{diff} \cdot \mathbf{n} d\mathbf{S} = \iiint_{V_f} \nabla J_{diff} dv_f = \nabla \iiint_{V_f} J_{diff} dv_f = \nabla (J_{diff} A_f dz) \quad (71)$$

By substituting the equations derived above into Eq. 67 we obtain

$$\frac{\partial (c_f A_f)}{\partial t} + \frac{\partial (c_f A_f v)}{\partial z} = \frac{\partial (c_s A_s)}{\partial t} + \frac{\partial (J_{diff} A_f)}{\partial z} \quad (72)$$

By defining $A_f = A \cdot \phi$, $A_s = A \cdot (1 - \phi)$ and $u = V \cdot \phi$, and assuming that A is constant, the above equation becomes

$$\frac{\partial (c_f \phi)}{\partial t} + \frac{\partial (c_f u)}{\partial z} = \frac{\partial (c_s (1 - \phi))}{\partial t} + \frac{\partial (J_{diff} \phi)}{\partial z} \quad (73)$$

By assuming that $\frac{\partial \phi}{\partial t} = 0$ and expanding J_{diff} , we get

$$\frac{\partial c_f}{\partial t} + \frac{1}{\phi} \frac{\partial (c_f u)}{\partial z} = \frac{(1 - \phi) \partial c_s}{\phi \partial t} + \frac{1}{\phi} \frac{\partial}{\partial z} \left(D_e^M \frac{\partial c_f}{\partial z} \right) \quad (74)$$

The equation can be further simplified if $\frac{\partial u}{\partial z} = \frac{\partial \phi}{\partial z} = D_e^M = 0$, which corresponds to the assumptions of constant velocity along the bed (which might be a case of isothermal and low-Mach number flow), constant porosity (which comes from the assumption of constant area for both solid and fluid phase) and no radial diffusion.

$$\frac{\partial c_f}{\partial t} + \frac{u}{\phi} \frac{\partial c_f}{\partial z} = \frac{1 - \phi}{\phi} \frac{\partial c_s}{\partial t} \quad (75)$$

The Eq. 75 is equivalent to the equation presented by Reverchon [1].

A.2.3. Momentum conservation

Similarly to mass conservation, momentum conservation is derived for inviscid fluid with no body forces

$$\frac{\partial}{\partial t} \iiint_{V_f} (\rho_f v_z) dV_f + \iint_S (\rho_f v_z \mathbf{V}) \cdot d\mathbf{S} = \iint_S (PdS)_z \quad (76)$$

where V_z is the z component of the velocity.

We the momentum conservation to the shaded control volume in Fig. 9, the integrals on the left side are evaluated in the same manner as discussed above in the regard to the continuity equation. That is,

$$\frac{\partial}{\partial t} \iiint_{V_f} (\rho_f v_z) dV_f = \frac{\partial}{\partial t} (\rho_f v A_f dz) \quad (77)$$

equation
and

$$\iint_S (\rho_f v_z \mathbf{V}) dS = -\rho_f v^2 + (\rho_f + d\rho_f)(v + dv)^2 (A + dA) \quad (78)$$

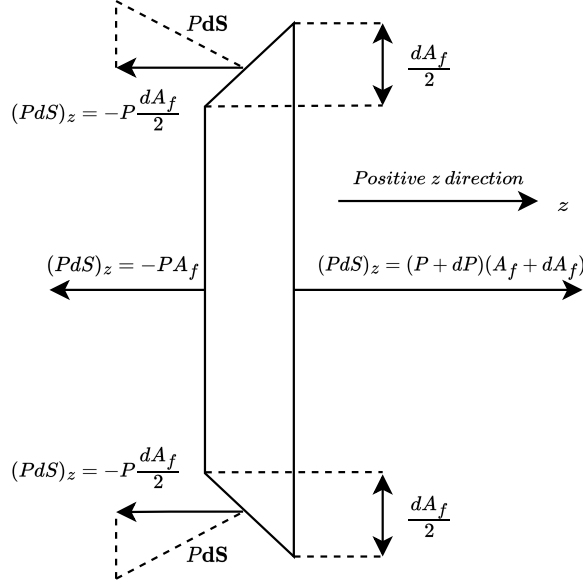


Figure 10: The forces in the z direction acting on the control volume

The evaluation of the pressure force term on the right side of Eq. 76 can be understood based on the Fig. 10. Here, the z components of the vector $Pd\mathbf{S}$ are shown on all four sides of the control volume. Remember that $d\mathbf{S}$ is assumed to point away from the control volume; hence any z component $(PdS)_z$ that acts toward the left (in the negative z direction) is a negative quantity. Any z component that acts toward the right (in the positive z direction) is a positive quantity. Also note that the z component of $Pd\mathbf{S}$ acting on the top and the bottom inclined faces of the control volume in Fig. 10 can be expressed as the pressure P acting on the component of the inclined is projected perpendicular to the z direction, $dA_f/2$; hence, the contribution of each inclined face (top or bottom) to the pressure integral in Eq. 76 is $-P(dA_f/2)$. All together, the right-hand side of Eq. 76 is expressed as follows:

$$\iint (PdS)_z = -PA_f + (P + dP)(A + dA_f) - 2P \frac{dA_f}{2} \quad (79)$$

Substituting Eqs. 77 to 79 into Eq. 76, we have

$$\begin{aligned} \frac{\partial}{\partial t} (\rho_f v A_f dz) - \rho_f v^2 A_f + (\rho_f + d\rho_f)(v + dv)^2 (A_f + dA_f) \\ = PA_f - (P + dP)(A + dA_f) + PdA_f \end{aligned} \quad (80)$$

Canceling like terms and ignoring products of differentials, the equation above becomes in the limit dz becoming very small

$$\frac{\partial}{\partial t} (\rho_f v A_f dz) + d(\rho_f v^2 A_f) = -AdP \quad (81)$$

Dividing the above equation by dz and taking the limit as dz goes to zero, we obtain

$$\frac{\partial (\rho_f v A_f)}{\partial t} + \frac{\partial (\rho_f v^2 A_f)}{\partial z} = -A_f \frac{\partial P}{\partial z} \quad (82)$$

The Eq. 82 can be expanded further by assuming that $A_f = \mathbf{A}\phi$

$$\frac{\partial (\rho_f v \mathbf{A}\phi)}{\partial t} + \frac{\partial (\rho_f v^2 \mathbf{A}\phi)}{\partial z} = -\mathbf{A}\phi \frac{\partial P}{\partial z} \quad (83)$$

The equation can be further simplified by assuming that the cross-section of an extractor \mathbf{A} is constant and cancel out

$$\frac{\partial (\rho_f v \phi)}{\partial t} + \frac{\partial (\rho_f v^2 \phi)}{\partial z} = -\phi \frac{\partial P}{\partial z} \quad (84)$$

If the superficial velocity $u = \phi V$ is introduced, then the momentum conservation becomes

$$\frac{\partial (\rho_f u)}{\partial z} + \frac{\partial (\rho_f u^2 / \phi)}{\partial z} = -\phi \frac{\partial P}{\partial z} \quad (85)$$

Eq. 82 represents the conservative form of the momentum equation for the quasi-one-dimensional flow. The equivalent non-conservative form can be obtained by multiplying the continuity equation by v and subtracting it from Eq. 82

$$\frac{\partial (\rho_f v A_f)}{\partial t} - V \frac{\partial (\rho_f A_f)}{\partial t} + \frac{(\rho_f v^2 A_f)}{\partial z} - V \frac{(\rho_f v A_f)}{\partial z} = -A_f \frac{\partial P}{\partial z} \quad (86)$$

Expanding the derivatives on the left-hand side of the above equation and canceling like terms, gives

$$\rho_f A_f \frac{\partial v}{\partial t} + \rho_f A_f v \frac{\partial v}{\partial z} = -A_f \frac{\partial P}{\partial z} \quad (87)$$

Dividing the above equation by A_f the non-conservative form of the momentum can be obtained

$$\rho_f \frac{\partial v}{\partial t} + \rho_f v \frac{\partial v}{\partial z} = -\frac{\partial P}{\partial z} \quad (88)$$

The Eq. 88 is stylistically the same as the general momentum conservation for one-dimensional flow with no-body forces. The momentum equation can be expressed in terms of superficial velocity $u = v\phi$.

$$\rho_f \frac{\partial (u/\phi)}{\partial t} + \rho_f \frac{u}{\phi} \frac{\partial (u/\phi)}{\partial z} = -\frac{\partial P}{\partial z} \quad (89)$$

By expanding all the terms of the equation above, we get

$$\frac{\rho_f}{\phi} \frac{\partial u}{\partial t} + \rho_f u \frac{\partial \phi^{-1}}{\partial t} + \rho_f \frac{u}{\phi} \frac{\partial u}{\partial z} + \rho_f \frac{u}{\phi} \frac{\partial \phi^{-1}}{\partial z} = -\frac{\partial P}{\partial z} \quad (90)$$

If the bed is not compressible and doesn't change its properties during the batch, then $\frac{\partial \phi}{\partial t} = 0$

$$\frac{\rho_f}{\phi} \left(\frac{\partial u}{\partial t} + \frac{u}{\phi} \frac{\partial u}{\partial z} + u^2 \frac{\partial \phi^{-1}}{\partial z} \right) = -\frac{\partial P}{\partial z} \quad (91)$$

If the porosity is constant along an extractor, then the momentum conservation equation becomes

$$\frac{\rho_f}{\phi} \left(\frac{\partial u}{\partial t} + \frac{u}{\phi} \frac{\partial u}{\partial z} \right) = -\frac{\partial P}{\partial z} \quad (92)$$

The Eq. 92 represents the non-conservative form of the momentum equation for quasi-one-dimensional flow with no body forces and constant porosity.

A.2.4. Energy conservation

Let's consider the integral form of the energy equation for adiabatic flow with no body forces and no viscous effects

$$\frac{\partial}{\partial t} \iiint_{V_f} \rho_f \left(e_f + \frac{v^2}{2} \right) dV_f + \iint_S \rho_f \left(e_f + \frac{v^2}{2} \right) \mathbf{v} \cdot d\mathbf{S} = - \iint_S (P\mathbf{V}) \cdot d\mathbf{S} \quad (93)$$

Applied to the shaded control volume in Fig. 9, and keeping in mind the pressure forces shown in Fig. 10, Eq. 93 becomes

$$\begin{aligned} & \frac{\partial}{\partial t} \left[\rho_f \left(e_f + \frac{v^2}{2} \right) A_f dz \right] - \rho_f \left(e_f + \frac{v^2}{2} \right) v A_f \\ & + (\rho_f + d\rho_f) \left[e_f + de_f + \frac{(v+dv)^2}{2} \right] (V + \textcolor{red}{V}) (A_f + dA_f) \\ & = - \left[-PvA_f + (P+dP)(v+dv)(A_f+dA_f) - 2 \left(Pv \frac{dA_f}{2} \right) \right] \end{aligned} \quad (94)$$

Check fV

Neglecting products of differential and canceling like terms, the above equation becomes

$$\frac{\partial}{\partial t} \left[\rho_f \left(e_f + \frac{v^2}{2} \right) A_f dz \right] + d \left(\rho_f r_f A_f \right) + \frac{(\rho_f v^3 A_f)}{2} = -d(PA_f v) \quad (95)$$

or

$$\frac{\partial}{\partial t} \left[\rho_f \left(e_f + \frac{v^2}{2} \right) A_f dz \right] + d \left[\rho_f \left(e_f + \frac{v^2}{2} \right) v A_f \right] = -d(PA_f v) \quad (96)$$

Taking the limit as dz approaches zero, the equation above becomes the following partial differential equation

$$\frac{\partial [\rho_f (e_f + v^2/2) A]}{\partial t} + \frac{\partial \rho_f (e_f + v^2/2) v A_f}{\partial z} = - \frac{\partial (PA_f v)}{\partial z} \quad (97)$$

Equation 97 is the conservation form of the energy expressed in terms of the total energy $e + v^2/2$, appropriate for unsteady, quasi-one-dimensional flow. The energy equation can be expressed in terms of internal energy if Eq. 82 is multiplied by v and then subtracted from Eq. 97

$$\frac{\partial (\rho_f e_f A_f)}{\partial t} + \frac{\partial (\rho_f e_f v A_f)}{\partial z} = -P \frac{\partial A_f v}{\partial z} \quad (98)$$

The equation above is the conservation form of the energy equation expressed in terms of internal energy e_f suitable for quasi-one-dimensional flow. The non-conservative for is then obtained by multiplying the continuity equation 63, by e_f and subtracting it from 98, yielding

$$\rho_f A_f \frac{\partial e_f}{\partial t} + \rho_f A_f v \frac{\partial e_f}{\partial z} = -P \frac{\partial (A_f v)}{\partial z} \quad (99)$$

Expanding the right-hand side and dividing by A_f , the above equation becomes

$$\rho_f \frac{\partial e_f}{\partial t} + \rho_f v \frac{\partial e_f}{\partial z} = -P \frac{v}{A_f} \frac{\partial A_f}{\partial z} \quad (100)$$

or

$$\rho_f \frac{\partial e_f}{\partial t} + \rho_f v \frac{\partial e_f}{\partial z} = -P \frac{\partial v}{\partial z} - PV \frac{\partial (\ln A_f)}{\partial z} \quad (101)$$

Equation 101 is the non-conservative form of the energy equation expressed in terms of internal energy, appropriate to unsteady quasi-one-dimensional flow. The reason for obtaining the energy equation in the form of Eq. 101 is that, for a calorically perfect gas, it leads directly to a form of the energy equation in terms of temperature T .

A.3. Bayes theorem

As discussed by Himmelblau [24], the Bayesian approach to estimation makes use of prior information. Such prior knowledge can come from theoretical considerations, from the results of previous experiments, or from assumptions by the experimenter. Typically, a Bayesian approach assumes a prior probability distribution of an unknown parameter θ in some parameter space Θ . The distribution is updated by using Bayes' rule to obtain the posterior probability distribution.

Consider a set of events or outcomes, A_1, A_2, \dots, A_n , and some other event B . Bayes' theorem states that the probability that event A_i will occur, given that event, B has already occurred, which will be denoted by $P\{A_i|B\}$, is equal to the product of the probability that A_i will occur regardless of whether B will take place and the probability that B will occur, given that A_i has already taken place, divided by the probability of the occurrence of B :

$$P\{A_i|B\} = \frac{P\{B|A_i\}P\{A_i\}}{P\{B\}} \quad (102)$$

Further, if all events comprising the set $\{A_i\}$ are included in A_1, A_2, \dots, A_n , then

$$P\{A_i|B\} = \frac{P\{B|A_i\}P\{A_i\}}{\sum_{i=1}^n P\{B|A_i\}P\{A_i\}} \quad (103)$$

We can interpret these symbols as follows:

1. $P\{A_i\}$ is a measure of our degree of belief that event A_i will occur or that hypothesis A_i is true prior to the acquisition of additional evidence that may alter the measure. $P\{A_i\}$ is denoted the *prior probability*.
2. $P\{A_i|B\}$ is a measure of our degree of belief that event A_i will occur or that hypothesis A_i is true, given additional evidence B pertinent to the hypothesis. $P\{A_i|B\}$ is termed the *posterior probability*.
3. $P\{B|A_i\}$ denotes the likelihood that event B will occur, given that event A_i is true. $P\{B|A_i\}$ is a conditional probability, interpreted in the Bayesian framework as a likelihood, $L(A_i|B)$.

For a continuous variables, Bayes' theorem can be more conveniently expressed in terms of the probability density function rather than the probabilities themselves. Equation 103 can be expressed in terms of a set of observed values of the random variable X , the experimental data \mathbf{x} and unknown parameter θ as

$$p(\theta|X=\mathbf{x}) = p(\theta|\mathbf{x}) = \frac{L(\theta|\mathbf{x})p(\theta)}{\int_{-\infty}^{+\infty} L(\theta|\mathbf{x})p(\theta)d\theta} \quad (104)$$

where

$p(\theta|\mathbf{x})$ = the posterior probability density function for θ ; it includes knowledge of the possible values of θ gained from the experimental data \mathbf{x}

$p(\theta)$ = the prior probability density function for θ (before the experiment in which \mathbf{x} was observed)

$L(\theta|\mathbf{x}) = p(\mathbf{x}|\theta)$ = the probability density function termed the likelihood function of θ given \mathbf{x}

The denominator in Equation 104 is a normalizing factor chosen so that the integration of the posterior distribution is unit, i.e., $\int_{-\infty}^{+\infty} p(\theta|\mathbf{x}) d\theta = 1$. By taking into account the law of total probability, the denominator can be written as

$$\int_{-\infty}^{+\infty} p(\mathbf{x}|\theta) p(\theta) d\theta = p(\mathbf{x}) \quad (105)$$

If the prior distribution is a uniform distribution, that is the prior distribution is a constant, the Equation 104 reduces to

$$p(\theta|\mathbf{x}) = \frac{L(\theta|\mathbf{x})}{\int_{-\infty}^{+\infty} L(\theta|\mathbf{x}) d\theta} \quad (106)$$

If the prior knowledge concerning a postulated event or hypothesis is poor, the posterior probability is largely or entirely determined by the likelihood function, that is, by the additional accumulated evidence for which the likelihood function acts as a mathematical expression. If prior knowledge outweighs recent evidence, however, then the posterior probability is determined almost solely by the prior probability.

In the application of tests and the design of experiments, certain definitions and rules concerning probability are needed and are listed below.

1. It follows from the frequency theory of probability that: $0 \leq P \leq 1$
2. If the probability of occurrence of one event A depends on whether or not event B has occurred, the two events are termed *dependent*; if the probability of occurrence of event A does not depend on the occurrence of B or the reverse, the two events are *independent*.
3. **Addition Rule** If A_1, A_2, \dots, A_n are mutually exclusive events, i.e., cannot occur at the same time, the probability of occurrence of just one of the events is equal to the sum of the probabilities of each A_i :

$$P(A_1 \text{ or } A_2 \dots \text{ or } A_n) = \sum_{i=1}^n P(A_i) \quad (107)$$

Very often we let

$$\sum_{i=1}^n P(A_i) = 1 \quad (108)$$

Also, if each event is equiprobable so that $P(A_i) = q$,

$$\sum_{i=1}^n q = nq = 1 \quad \text{or} \quad q = \frac{1}{n} = P(A_i) \quad (109)$$

In set theory, mutually exclusive events have no points in common. The union of the sets which represents the set of all elements that belong to

$$P(A_1 \cup A_2 \cup \dots \cup A_n) = P(A_1) + P(A_2) + \dots + P(A_n) \quad (110)$$

4. **Multiplication Rule** If A and B are *independent* event

$$P(A \text{ and } B) = P(A)P(B) \quad (111)$$

In set theory the intersection of A and B is the set of all elements that belong to A and B :

$$P(A \cap B) = P(A)P(B) \quad (112)$$

If the A and B are *dependent* events,

$$P(A|B) = \frac{P(A \cap B)}{P(B)} \quad (113)$$

where the symbol $P(A|B)$ means "probability of A given B ". As a corollary,

$$P(A \cap B) = P(B)P(A|B) \quad (114a)$$

$$= P(A)P(B|A) \quad (114b)$$

Two kinds of probabilities enter Equation 114a (or Equation 114b): the absolute probability of event B (or A) irrespective of whether or not A (or B) has occurred, and that the conditional probability of event A (or B) computed on the assumption that B (or A) has occurred. Equation 112 or 113 is special case of Equation 114a or 114b, because if the event are independent $P(A|B) = P(A)$. For the case of many event, Equation 112 can be expanded to

$$\begin{aligned} P(A_1 \text{ and } A_2 \text{ and } \dots \text{ and } A_n) &= P(A_1) \cdot P(A_2) \cdot \dots \cdot P(A_n) \\ &= \prod_{i=1}^n P(A_i) \end{aligned} \quad (115)$$

5. Another useful relationship for events that are not mutually exclusive is

$$P(A) + P(B) - P(A \cap B) = P(A \cup B) \quad (116)$$

A.4. Cardano's Formula

Following the work of Gmehling et al. [20], a cubic equation of state can be written a following form

$$Z^3 + UZ^2 + SZ + T = 0 \quad (117)$$

with Z as the compressibility factor. Using Cardano's formula, this type of equation can be solved analytically.

With the abbreviations

$$P = \frac{3S - U^2}{3} \quad Q = \frac{2U^3}{27} - \frac{US}{3} + T$$

the discriminant can be determined to be

$$D = \left(\frac{P}{3}\right)^3 + \left(\frac{Q}{2}\right)^2 \quad (118)$$

For $D > 0$, the equation of state has one real solution:

$$Z = \left[\sqrt{D} - \frac{Q}{2} \right]^{1/3} - \frac{P}{3 \left[\sqrt{D} - \frac{Q}{2} \right]^{1/3}} - \frac{U}{3} \quad (119)$$

For $D < 0$, there are three real solutions. With the abbreviations

$$\Theta = \sqrt{-\frac{P^3}{27}} \quad \Phi = \arccos\left(\frac{-Q}{2\Theta}\right)$$

they can be written as

$$Z_1 = 2\Theta^{1/3} \cos\left(\frac{\Phi}{3}\right) - \frac{U}{3} \quad (120)$$

$$Z_2 = 2\Theta^{1/3} \cos\left(\frac{\Phi}{3} + \frac{2\pi}{3}\right) - \frac{U}{3} \quad (121)$$

$$Z_3 = 2\Theta^{1/3} \cos\left(\frac{\Phi}{3} + \frac{4\pi}{3}\right) - \frac{U}{3} \quad (122)$$

The largest and the smallest of the three values correspond to the vapor and to the liquid solutions, respectively. The middle one has no physical meaning.

A.5. Initial and boundary conditions

It is assumed that the solvent is free of solute at the entrance of the extractor and that all the solid particles have the same initial solute content c_s^0 . It is considered that the initial temperature of the extractor in every place is the same and described by h^0 .

During the preparation period, when the operating conditions are achieved, the solute diffuses to the fluid phase in contact with the solid particles. Later, the solute in the fluid phase is partially moved (if the pressure increase in the system, the pump cause the movement of the fluid, even if the outlet valve is closed) to the region where there is no solid phase. As a result, the distribution of solutes mass in the fluid phase is assumed to not be uniform, and described by an arbitrary function $H = H(z)$. Some conclusions can be drawn from the analysis of the initial part of each yield curve. It can be noticed that each curve at the beginning has a curvature, which is not linear. In a general sense, it can be said that a quadratic function could approximate the initial part of each extraction curve. A function that, after integration, gives a quadratic-like result is a straight line. Based on that observation, the solute concentration in the fluid phase is assumed to be linearly distributed. The solute concentration is assumed to be zero at the outlet and reach the maximum at the beginning of the fixed bed. The graphical representation of the solute concentration in the fluid phase is shown in Figure 11.

As presented in Figure 11, the function H is defined as

$$H(z) = -m(z - b) + a \quad (123)$$

The function H can be integrated over the integral from a to b to describe the total amount of solute in the fluid phase (S) in that interval.

$$\int_a^b H(z) dz = S \quad (124)$$

If parameter a describe the beginning of the fixed bed, then $a = 0$. Similarly, b can be defined as the end of the fixed bed, then $b = L$. On range from $a = 0$ to $b = L$ the result of this integral is know, and it is equal to

$$S = m_{fluid}^0 = \int_{a=0}^{b=L} -m(x - L) dz \quad (125)$$

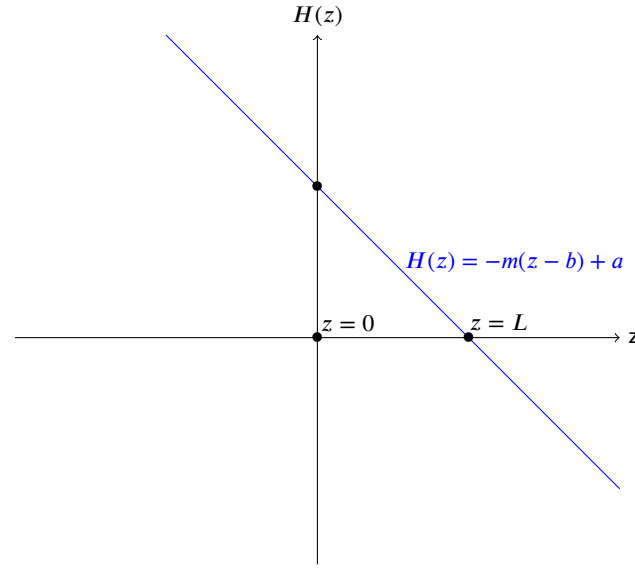


Figure 11: The linear distribution of the solute concentration in the fluid phase

where m_{fluid}^0 is the total mass of solute in the fluid phase and N is the number of intervals between a and b . The right-hand side of the above equation can be evaluated

$$\begin{aligned} \int_{a=0}^{b=L} -m(x - L) dz &= -m \left(\int_{a=0}^{b=L} z dz - \int_{a=0}^{b=L} L dz \right) \\ &= -m \left(\frac{z^2}{2} \Big|_0^L - Lz \Big|_0^L \right) \\ &= -m \left(\frac{L^2}{2} - L^2 \right) \\ &= m \frac{L^2}{2} \end{aligned} \quad (126)$$

The parameter m can be obtained by equating the left-hand side of Equation 125 and the right-hand side of Equation 126

$$m_{fluid}^0 = m \frac{L^2}{2} \rightarrow m = \frac{2m_{fluid}^0}{L^2} \quad (127)$$

As can be seen from the above equation, the linearly distributed concentration of solute in the fluid phase can be fully determined if parameters a , b and m_{fluid}^0 are know. m_{fluid}^0 can be obtained from the total mass of the solute in the system at the initial time m_{total}^0 and the partition factor τ .

$$\tau = \frac{\text{total mass of solute in the fluid phase}}{\text{total mass of solute in the system}} = \frac{m_{fluid}^0}{m_{total}^0}$$

The initial conditions can be summarized as

$$\begin{aligned} c_f(t = 0, z) &= H(z) \\ c_s(t = 0, z) &= c_{s0} \\ h(t = 0, z) &= h_0 \end{aligned}$$

A.6. Solid density measurement

$$\rho_s^{ave} = \frac{1.2585 + 1.2582 + 1.2561 + 1.2546 + 1.2555}{5} = 1.25658 [g/cc]$$



QUANTACHROME CORPORATION
Upvc 1200e V5.05
Analysis Report

Tue Oct 11 15:10:37 2022
User ID: OLIW

Sample Parameters
Sample ID: T5
Weight: 55.5411 g
Description: coal
Comment: Powder

Analysis Parameters
Cell Size - Large
V Added - Large: 80.8546 cc
V Cell: 149.7915 cc
Analysis Temperature: 27.9 C
Target Pressure: 131.0 kPa
Type of gas used: Helium
Equilibration Time: Auto
Flow Purge: 1.0 min.
Maximum Runs: 5
Number Of Runs Averaged: 5
Deviation Requested: 0.0050 %

Analysis Results
Deviation Achieved: 0.1135 %
Average Volume: 44.1994 cc
Volume Std. Dev.: 0.0542 cc
Average Density: 1.2566 g/cc
Density Std. Dev.: 0.0015 g/cc
Coefficient of Variation: 0.1227 %

Run Data		
RUN	VOLUME (cc)	DENSITY (g/cc)
1	44.1311	1.2585
2	44.1423	1.2582
3	44.2163	1.2561
4	44.2709	1.2546
5	44.2365	1.2555

Figure 12: The result of solid density measurement

A.7. Porosity Calculations

To calculate the porosity of the solid particles ϕ , we can use the following formula:

$$\phi = \frac{\text{Volume with pores} - \text{Volume without pores}}{\text{Volume with pores}} \quad (129)$$

We can first calculate the volume, excluding the pore space, using the true density of the material:

$$\text{Volume excluding pores} = \frac{\text{mass}}{\text{true density}} = \frac{1 \text{ kg}}{1256.58 \text{ kg/m}^3} 0.0007958 \text{ m}^3 \quad (130)$$

We can then use the volume of the solid particles, including the pore space and the volume excluding the pore space, to calculate the porosity:

$$\phi = \frac{\text{Volume with pores} - \text{Volume without pores}}{\text{Volume with pores}} \quad (131)$$

Since 1 kg of solid particles occupies 1.6 L, which is equivalent to 0.0016 m³, we have:

$$\phi = \frac{0.0016 \text{ m}^3 - 0.0007958 \text{ m}^3}{0.0016 \text{ m}^3} = 0.5026 \quad (132)$$

Therefore, the porosity of the solid particles is 0.5026 or 50.26%. This means that half of the total volume of the solid particles is made up of pore space.