

# Lab experiments start next week!

- To do **this week**:

- Experiments in wks 6,7, 9, 10, **2 – 5 pm**
  - Python returns in wks 11, 12, 13, **1 – 4 pm**
- Sign up for your first lab experiment ASAP
  - coordinate with preferred lab partner, if you have one
- Read first chapter of the lab notes
- Do the pre-lab reading for your first experiment and answer the questions
  - that means *before* you show up to the lab
  - your lab supervisor will check your answers early in the lab session

# Coupled SHM – Examples

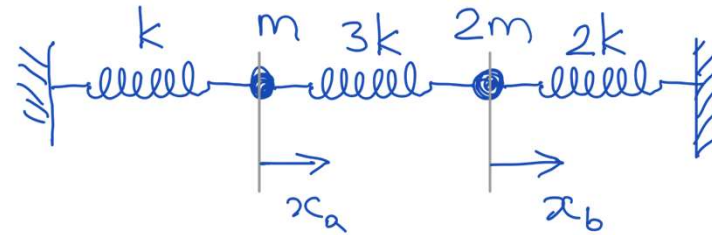
Prof David Spence

Based on slides provided by Prof David Spence

# Example – Exam 2019

## Question 1 (20 marks).

Two masses  $m$  and  $2m$  lie on a horizontal frictionless surface and are connected to each other and to a pair of fixed vertical walls by springs with spring constants  $k$ ,  $3k$ , and  $2k$  as illustrated in the figure at right. The displacements of the masses from their equilibrium positions are  $x_a$  and  $x_b$ , respectively.



- a) (4 marks) Write down expressions for the force on:
- (i)  $m$  when  $x_a = 0$
  - (ii)  $m$  when  $x_b = 0$
  - (iii)  $2m$  when  $x_a = 0$
  - (iv)  $2m$  when  $x_b = 0$
- b) (4 marks) Hence show that the displacements of the two masses from their equilibrium positions  $x_a$  and  $x_b$  obey the coupled equations:

$$\ddot{x}_a = -\frac{4k}{m} x_a + \frac{3k}{m} x_b$$

$$\ddot{x}_b = \frac{3k}{2m} x_a - \frac{5k}{2m} x_b$$

- c) (4 marks) Use the substitutions  $x_a = Ae^{i\omega t}$  and  $x_b = Be^{i\omega t}$  to show that the normal mode frequencies are

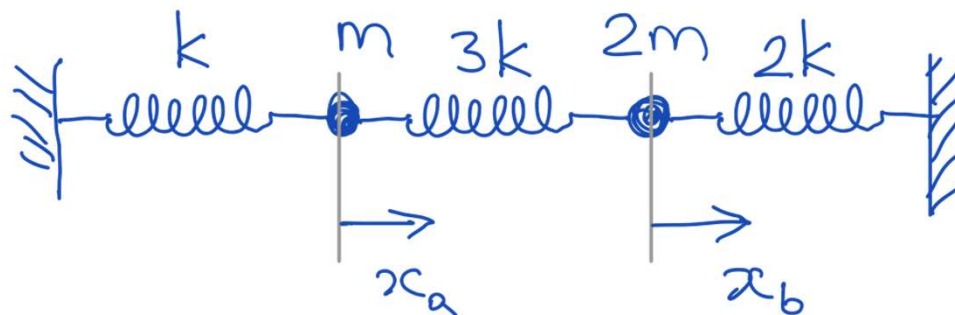
$$\omega_1 = \sqrt{\frac{k}{m}} \quad \text{and} \quad \omega_2 = \sqrt{\frac{11k}{2m}}$$

- d) (4 marks) Calculate the amplitude ratio  $A/B$  for each of the two normal modes.
- e) (4 marks) Consider the lowest-frequency mode. Explain how the springs are successively stretched and compressed during the oscillations and use this information to explain why the mode frequency is  $\sqrt{k/m}$ .

# Example – Exam 2019

## Question 1 (20 marks).

Two masses  $m$  and  $2m$  lie on a horizontal frictionless surface and are connected to each other and to a pair of fixed vertical walls by springs with spring constants  $k$ ,  $3k$ , and  $2k$  as illustrated in the figure at right. The displacements of the masses from their equilibrium positions are  $x_a$  and  $x_b$ , respectively.



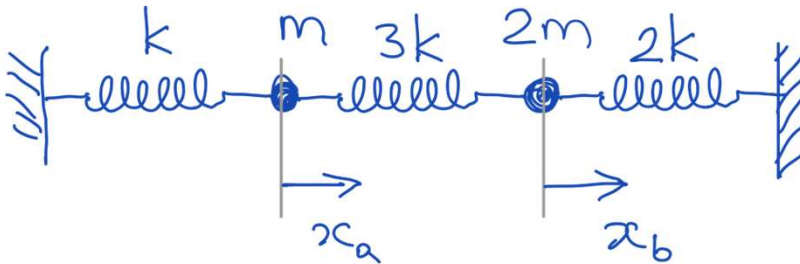
a) (4 marks) Write down expressions for the force on:

- |                         |          |                           |          |
|-------------------------|----------|---------------------------|----------|
| (i) $m$ when $x_a = 0$  | $3kx_b$  | (iii) $2m$ when $x_a = 0$ | $-5kx_b$ |
| (ii) $m$ when $x_b = 0$ | $-4kx_a$ | (iv) $2m$ when $x_b = 0$  | $3kx_a$  |

b) (4 marks) Hence show that the displacements of the two masses from their equilibrium positions  $x_a$  and  $x_b$  obey the coupled equations:

$$\begin{aligned} \text{From (a) (i) \& (ii), } m\ddot{x}_a &= 3kx_b - 4kx_a \Rightarrow \ddot{x}_a = -\frac{4k}{m}x_a + \frac{3k}{m}x_b \\ \text{(a) (iii) \& (iv), } 2m\ddot{x}_b &= 3kx_a - 5kx_b \Rightarrow \ddot{x}_b = \frac{3k}{2m}x_a - \frac{5k}{2m}x_b \end{aligned}$$

# Example – Exam 2019



$$\ddot{x}_a = -\frac{4k}{m}x_a + \frac{3k}{m}x_b$$

$$\ddot{x}_b = \frac{3k}{2m}x_a - \frac{5k}{2m}x_b$$

- c) (4 marks) Use the substitutions  $x_a = Ae^{i\omega t}$  and  $x_b = Be^{i\omega t}$  to show that the normal mode frequencies are

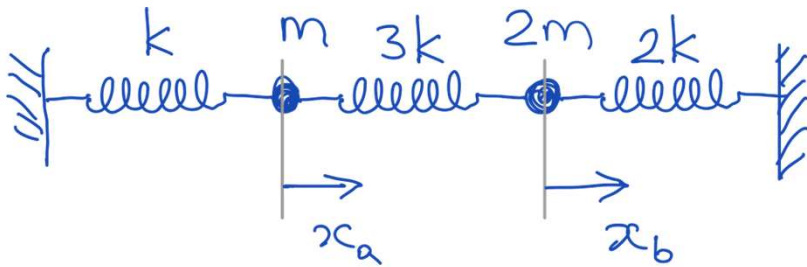
$$\omega_1 = \sqrt{\frac{k}{m}} \quad \text{and} \quad \omega_2 = \sqrt{\frac{11k}{2m}}$$

$$\begin{aligned} -\omega^2 A &= -\frac{4k}{m}A + \frac{3k}{m}B \\ -\omega^2 B &= \frac{3k}{2m}A - \frac{5k}{2m}B \end{aligned} \Rightarrow \odot = \begin{vmatrix} \omega^2 - \frac{4k}{m} & \frac{3k}{m} \\ \frac{3k}{2m} & \omega^2 - \frac{5k}{2m} \end{vmatrix}$$

$$\begin{aligned} &= \left(\omega^2 - \frac{4k}{m}\right)\left(\omega^2 - \frac{5k}{2m}\right) - \frac{3k}{m}\frac{3k}{2m} = \omega^4 - \frac{13k}{2m}\omega^2 + \frac{11}{2}\frac{k^2}{m^2} \\ &= \left(\omega^2 - \frac{k}{m}\right)\left(\omega^2 - \frac{11k}{2m}\right) \end{aligned}$$

$$\Rightarrow \omega_1 = \sqrt{\frac{k}{m}}, \quad \omega_2 = \sqrt{\frac{11k}{2m}}$$

# Example – Exam 2019



$$\ddot{x}_a = -\frac{4k}{m}x_a + \frac{3k}{m}x_b \quad x_a = Ae^{i\omega t} \text{ and } x_b = Be^{i\omega t}$$

$$\ddot{x}_b = \frac{3k}{2m}x_a - \frac{5k}{2m}x_b \quad \omega_1 = \sqrt{\frac{k}{m}} \text{ and } \omega_2 = \sqrt{\frac{11k}{2m}}$$

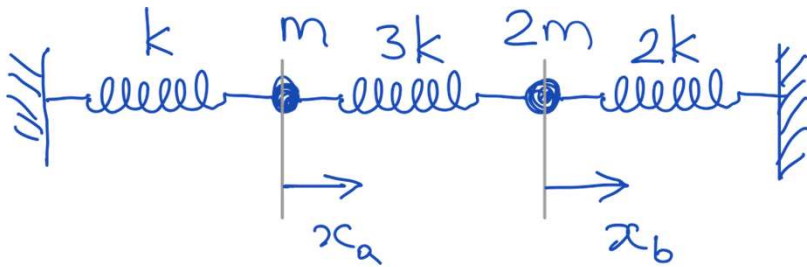
d) (4 marks) Calculate the amplitude ratio  $A/B$  for each of the two normal modes.

$$-\omega^2 A = -\frac{4k}{m}A + \frac{3k}{m}B \quad \omega_1 = \sqrt{\frac{k}{m}} \quad \frac{A}{B} = \frac{-3k/m}{\omega_1^2 - 4k/m} = \frac{-3k/m}{-3k/m} = +1$$

$$-\omega^2 B = \frac{3k}{2m}A - \frac{5k}{2m}B \quad \omega_2 = \sqrt{\frac{11k}{2m}} \quad \frac{A}{B} = \frac{-3k/m}{\omega_2^2 - 4k/m} = \frac{-3k/m}{3/2k/m} = -2$$



# Example – Exam 2019



$$\ddot{x}_a = -\frac{4k}{m}x_a + \frac{3k}{m}x_b$$

$$\ddot{x}_b = \frac{3k}{2m}x_a - \frac{5k}{2m}x_b$$

$$x_a = Ae^{i\omega t} \text{ and } x_b = Be^{i\omega t}$$

$$\omega_1 = \sqrt{\frac{k}{m}}$$

$$\frac{A}{B} = +1$$

$$\omega_2 = \sqrt{\frac{11k}{2m}}$$

$$\frac{A}{B} = -2$$

- e) (4 marks) Consider the lowest-frequency mode. Explain how the springs are successively stretched and compressed during the oscillations and use this information to explain why the mode frequency is  $\sqrt{k/m}$ .

For  $\omega_1$   $A = +B \Rightarrow x_a = x_b$ . Central spring is not stretched at all  
 $\therefore$  masses do not feel each others displacements

$\Rightarrow$   $m$  and  $2m$  behave as if isolated oscillators on springs

$\Rightarrow \omega = \sqrt{\frac{k}{m}}$  and  $\sqrt{\frac{2k}{2m}}$ , respectively.

# The loaded string

Prof David Spence

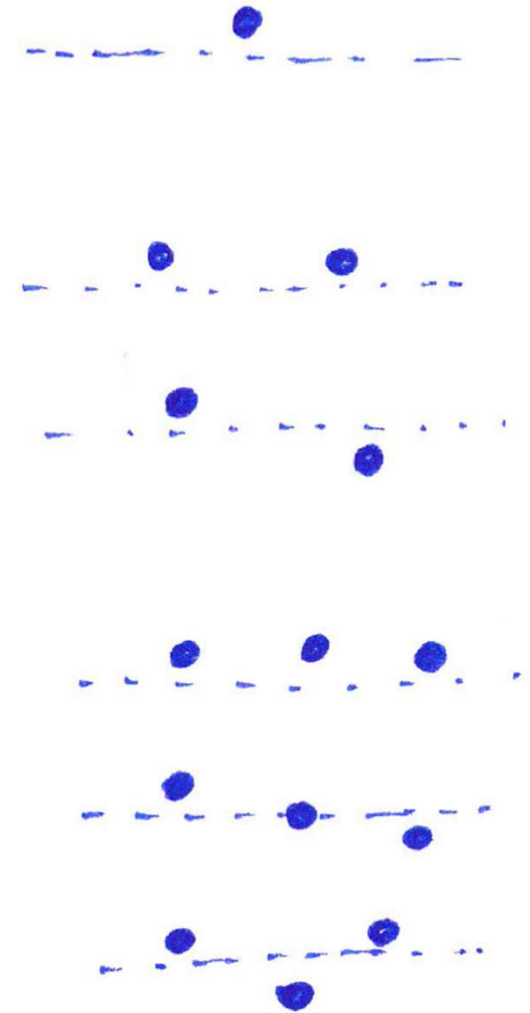
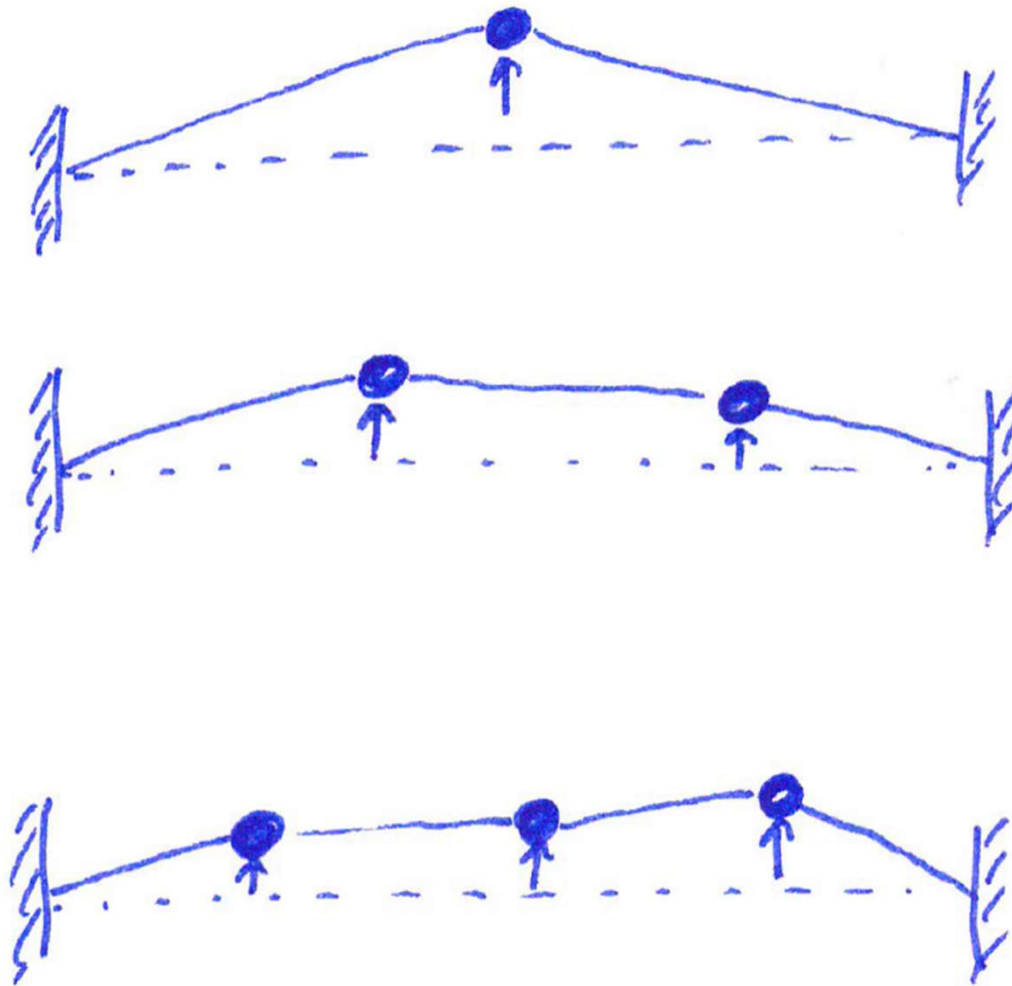


## Recall: Properties of normal modes

- For a normal mode
  - Oscillation is independent of other normal modes coordinates
  - Energy is not transferred to other normal modes
  - All oscillators have a fixed phase relation
  - All oscillators have fixed amplitude ratio
  - All oscillators share the same frequency
- $N$  oscillators  $\rightarrow$   $N$  normal modes
- Lowest frequency mode has all oscillators in phase
- Highest frequency mode has adjacent oscillators out of phase

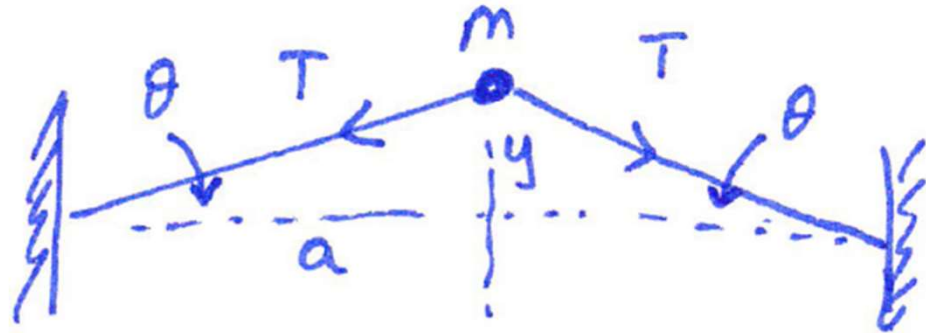
## Let's look at a 'loaded string'

- Light string under tension, loaded with point masses



# One mass

- Light string
- Point mass
- Constant tension
- Small deviation

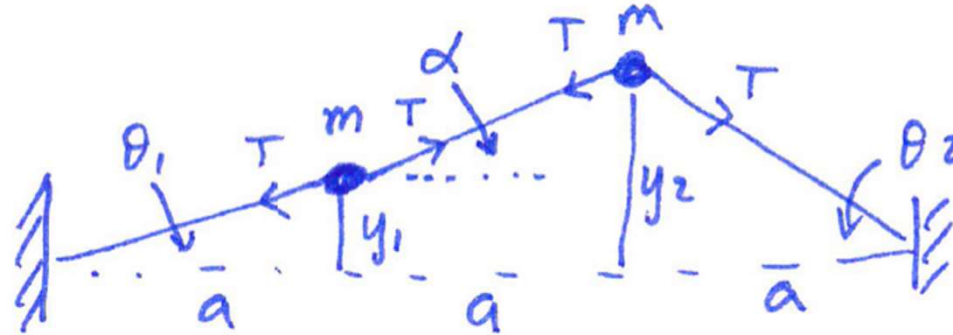


$$F = m\ddot{y} = -2T\sin\theta \approx -2T\tan\theta \approx -2Ty/a \quad \theta \ll 1$$

$$\ddot{y} + \frac{2T}{ma}y = 0$$

SHM with frequency  $\omega^2 = 2T/(ma)$

## Two masses...



left mass:  $m\ddot{y}_1 = -T \sin \theta_1 + T \sin \alpha \approx -T(y_1/a - (y_2 - y_1)/a)$

$$\ddot{y}_1 + (T/ma)(2y_1 - y_2) = 0$$

right mass:  $\ddot{y}_2 + (T/ma)(2y_2 - y_1) = 0$

Add:  $\ddot{y}_1 + \ddot{y}_2 + (T/ma)(y_1 + y_2) = 0$

Subtract:  $\ddot{y}_1 - \ddot{y}_2 + (3T/ma)(y_1 - y_2) = 0$

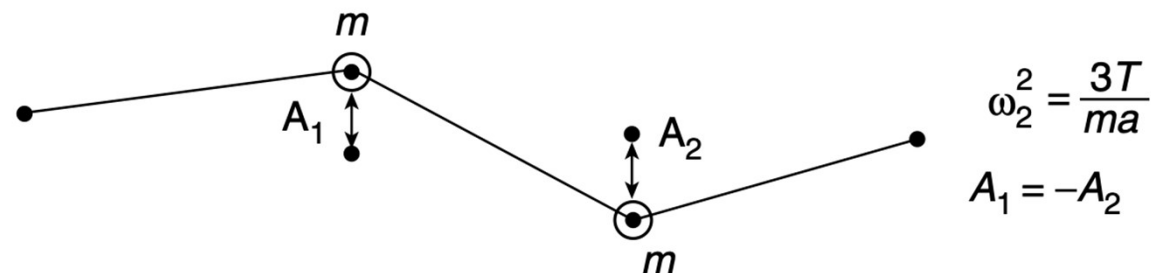
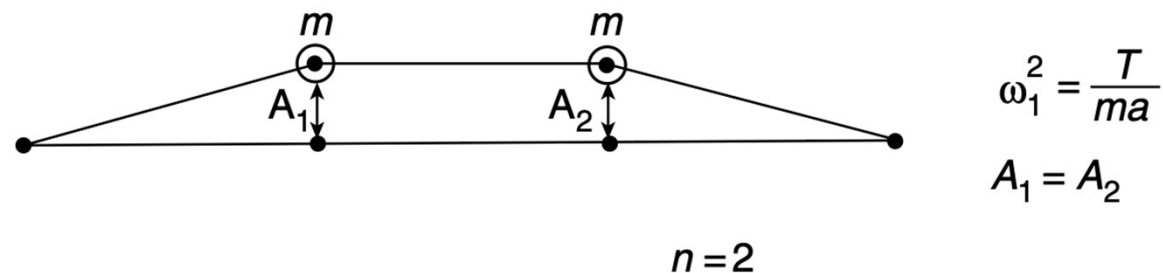
# Two masses have two normal modes\*

$$\ddot{y}_1 + \ddot{y}_2 + (T/ma)(y_1 + y_2) = 0$$

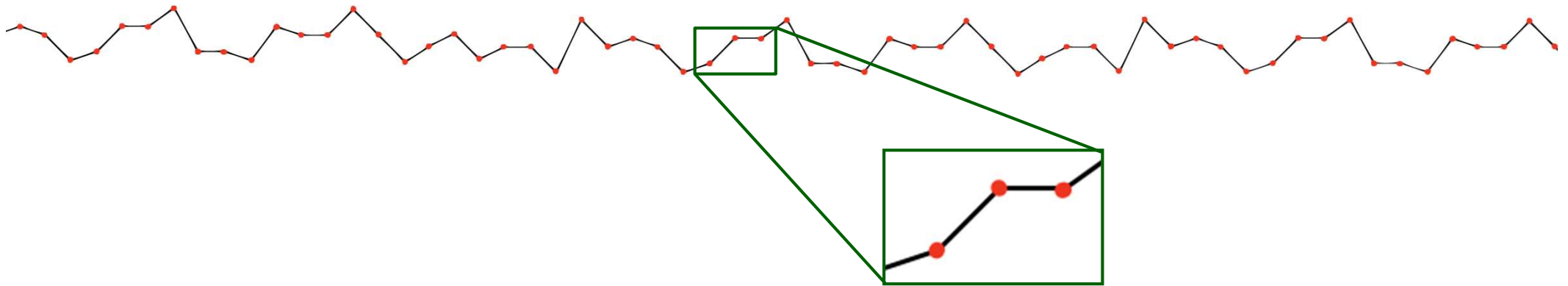
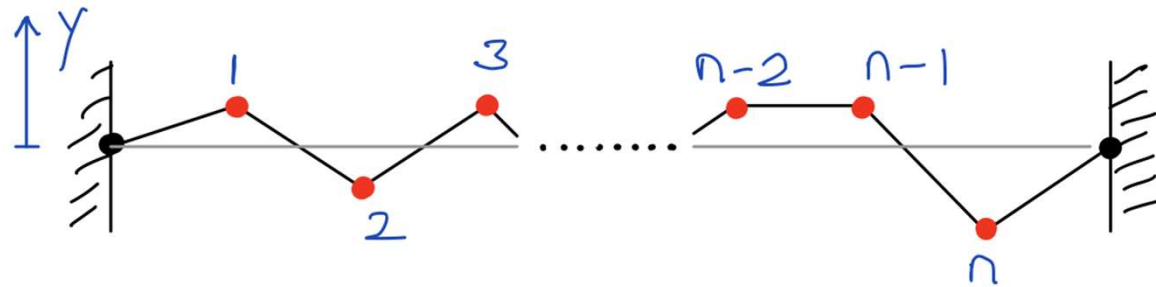
$$\ddot{y}_1 - \ddot{y}_2 + (3T/ma)(y_1 - y_2) = 0$$

Two normal modes:  $Y_1 = y_1 + y_2$  with  $\omega_1^2 = T/ma$  and  $A_1 = A_2$

$Y_2 = y_1 - y_2$  with  $\omega_2^2 = 3T/ma$  and  $A_1 = -A_2$

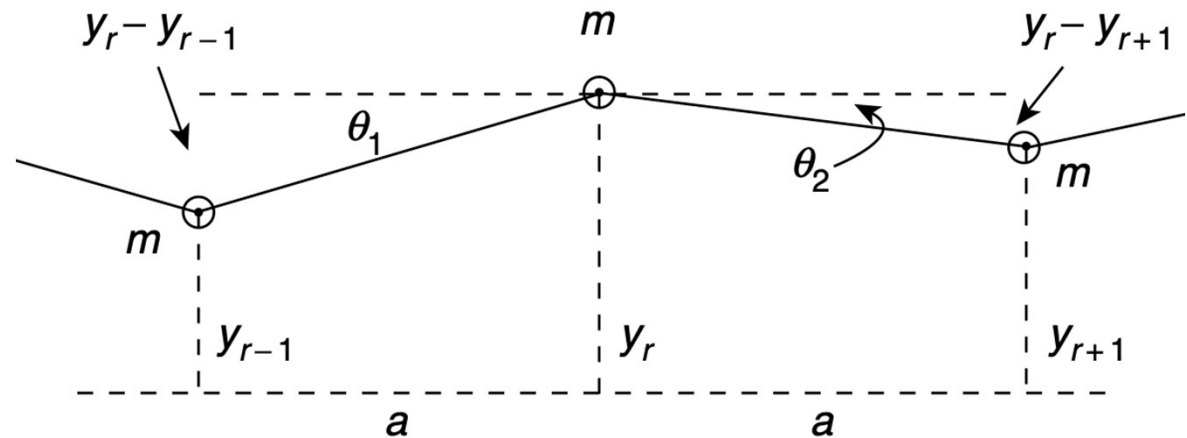
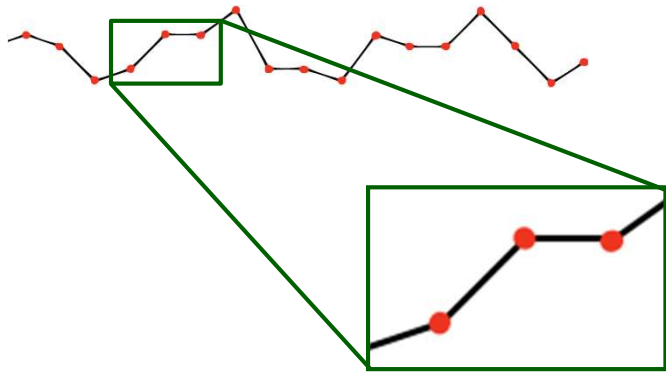


# Generalise to n masses (yikes!)



look at the  $r^{\text{th}}$  mass:  $\ddot{y}_r$  depends only on the positions  $y_r$ ,  $y_{r-1}$  and  $y_{r+1}$

# Generalise to n masses (yikes!)



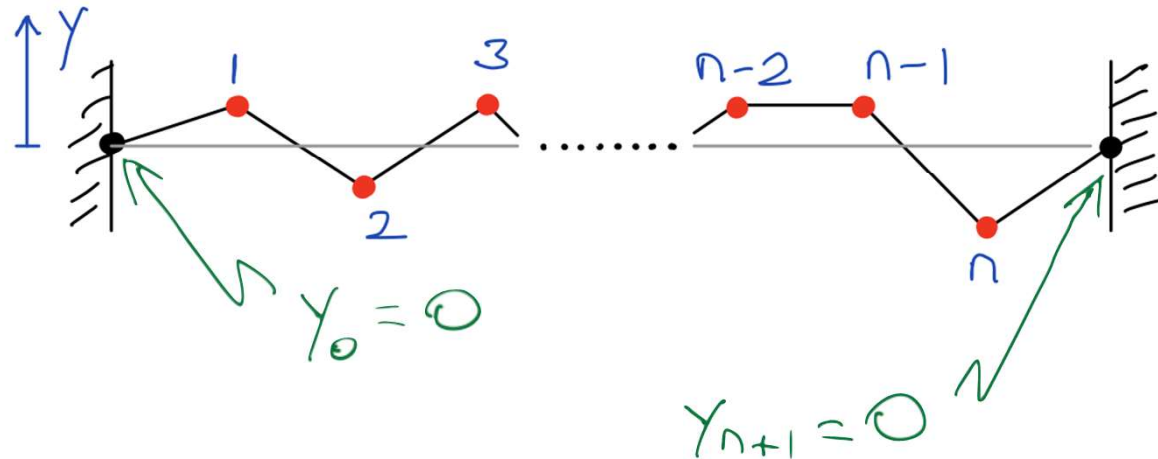
$$m\ddot{y}_r = -T \sin \theta_1 - T \sin \theta_2$$
$$\approx -T[(y_r - y_{r-1})/a + (y_r - y_{r+1})/a]$$

$$\ddot{y}_r + (T/ma)(2y_r - y_{r+1} - y_{r-1}) = 0 \quad r = 1, 2, \dots, n$$

note that the first and last masses ( $r = 1$  and  $r = n$ ) are special



$$\ddot{y}_r + (T/ma)(2y_r - y_{r+1} - y_{r-1}) = 0 \quad r = 1, 2, \dots, n$$



first and last masses depend on the left and right endpoints,  $y_0$  and  $y_{n+1}$

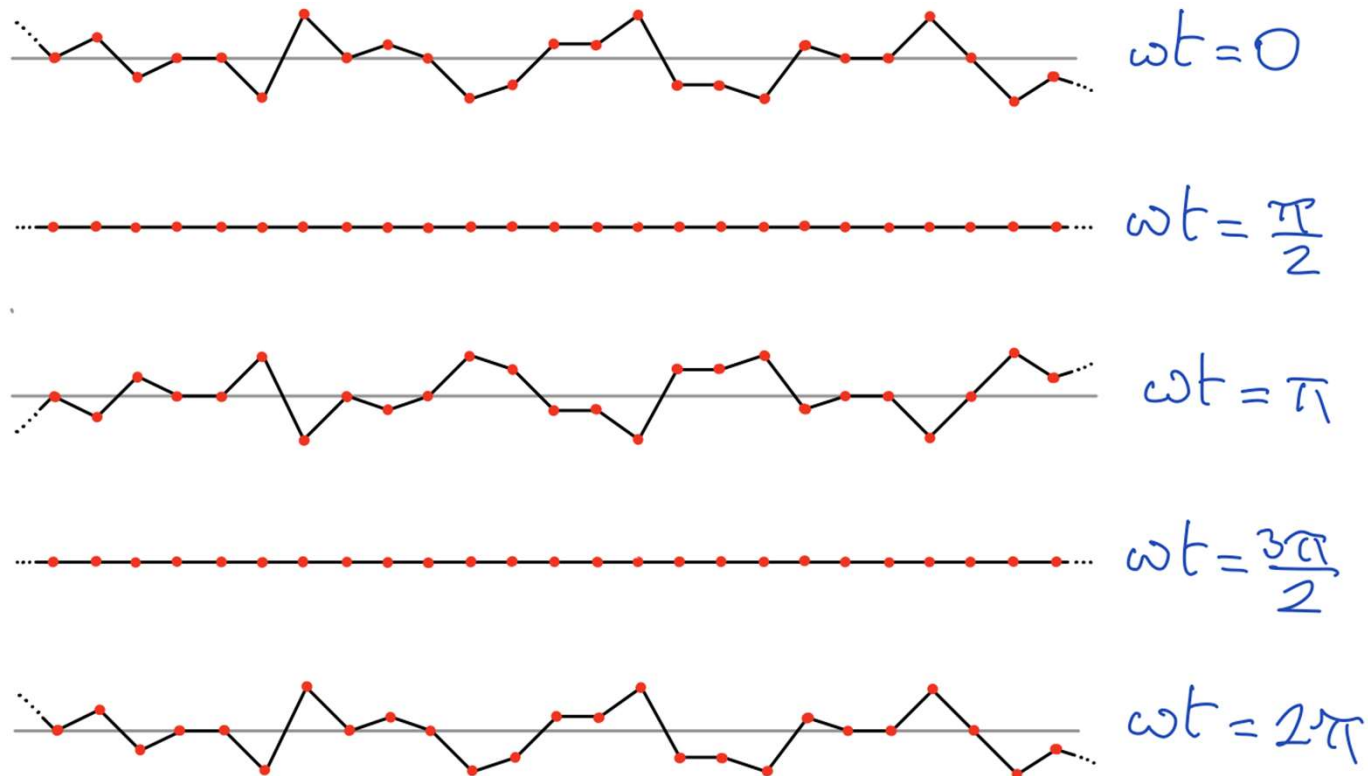
Boundary conditions:  $y_0 = y_{n+1} = 0$

$$\ddot{y}_r + (T/ma)(2y_r - y_{r+1} - y_{r-1}) = 0$$

$$y_0 = y_{n+1} = 0$$

$$r = 1, 2, \dots, n$$

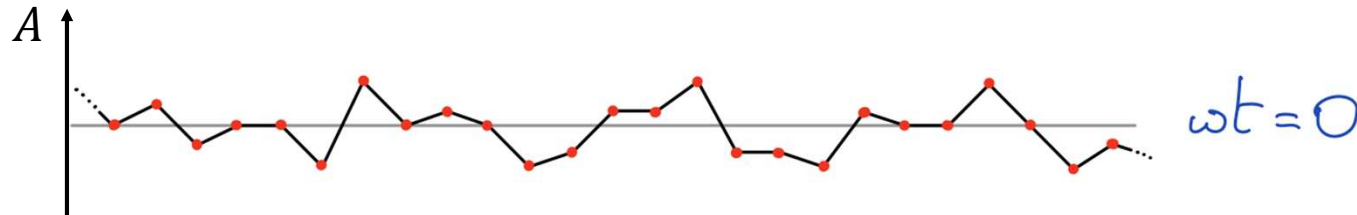
Look for normal modes:  $y_r = A_r e^{i\omega t}$ ,  $y_{r+1} = A_{r+1} e^{i\omega t}$ ,  $y_{r-1} = A_{r-1} e^{i\omega t}$



$$\ddot{y}_r + (T/ma)(2y_r - y_{r+1} - y_{r-1}) = 0 \quad y_0 = y_{n+1} = 0$$

$$r = 1, 2, \dots, n$$

Look for normal modes:  $y_r = A_r e^{i\omega t}$ ,  $y_{r+1} = A_{r+1} e^{i\omega t}$ ,  $y_{r-1} = A_{r-1} e^{i\omega t}$



$$-\omega^2 A_r + T/(ma)(2A_r - A_{r+1} - A_{r-1}) = 0$$

$$-A_{r-1} + \left(2 - \frac{\omega^2 ma}{T}\right) A_r - A_{r+1} = 0$$

$n$  equations for  $r = 1$  to  $r = n$ , with  $A_0 = A_{n+1} = 0$

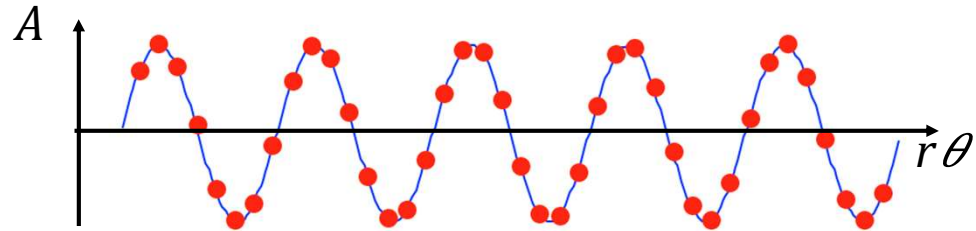
$$-A_{r-1} + \left(2 - \frac{\omega^2 ma}{T}\right) A_r - A_{r+1} = 0$$

$$\frac{A_{r+1} + A_{r-1}}{A_r} = \frac{2\omega_0^2 - \omega^2}{\omega_0^2} \text{ with } \omega_0^2 = T/ma$$

A moment of genius: look for modes with  $A_r = C e^{i(r\theta + \phi)}$  Note that C is real

Take real value to compare  
with physical coordinates:

$$\text{Re}[A_r] = a_r = C \cos(r\theta + \phi)$$



$$\frac{C e^{i(r+1)\theta + i\phi} + C e^{i(r-1)\theta + i\phi}}{C e^{ir\theta + i\phi}} = e^{i\theta} + e^{-i\theta} = 2 \cos \theta$$

$$e^{x+y} = e^x e^y$$

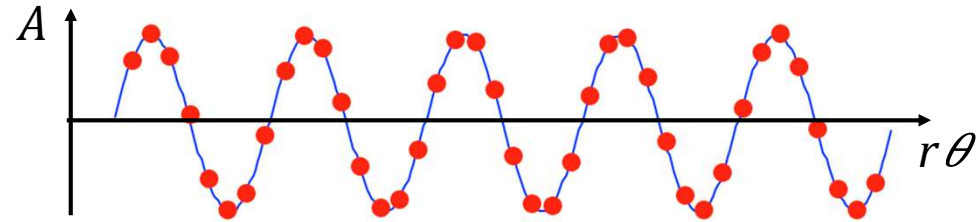
$$e^{x-y} = e^x / e^y$$

$$A_r = C e^{i(r\theta + \phi)} \text{ is a valid solution as long as } 2 \cos \theta = \frac{2\omega_0^2 - \omega^2}{\omega_0^2}$$

$$A_r = C e^{i(r\theta + \phi)}$$

$$\text{where } 2 \cos \theta = \frac{2\omega_0^2 - \omega^2}{\omega_0^2}$$

$$\text{Re}[A_r] = a_r = C \cos(r\theta + \phi)$$



$A_r$  = amplitude of oscillation of mass  $r$

$$y_r = A_r e^{i\omega t}$$

Boundary conditions:  $a_0 = a_{n+1} = 0$

$$a_0 = C \cos(\phi) = 0 \Rightarrow \phi = -\pi/2 \Rightarrow a_r = C \sin(r\theta)$$

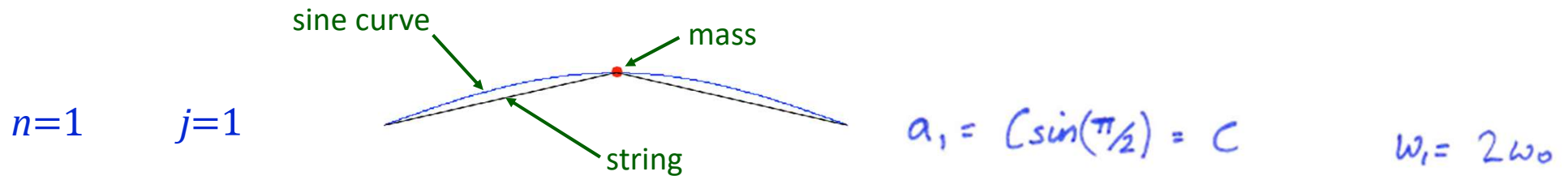
$$a_{n+1} = C \sin([n+1]\theta) = 0 \Rightarrow \theta = j\pi/(n+1), \quad j = 1, 2, \dots, n$$

$n$  solutions

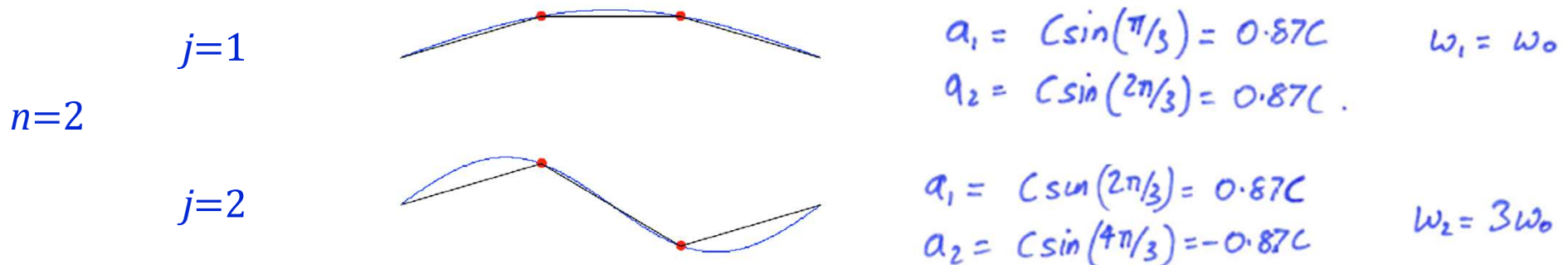
$$\therefore a_r = C \sin\left(\frac{j\pi r}{n+1}\right) \quad \text{and} \quad \omega_j^2 = 2\omega_0^2 \left[1 - \cos\left(\frac{j\pi}{n+1}\right)\right]$$

# Masses lie on a sine curve.....

$$\therefore a_r = C \sin\left(\frac{j\pi r}{n+1}\right) \quad \text{and} \quad \omega_j^2 = 2\omega_0^2 \left[1 - \cos\left(\frac{j\pi}{n+1}\right)\right] \quad j = 1, 2, 3, \dots, n$$



Masses are connected by straight string segments (any curvature implies infinite acceleration!)



Additional notes: <http://www.ectropy.info/scholar/node/the-loaded-string>

# They do!

$n=3$

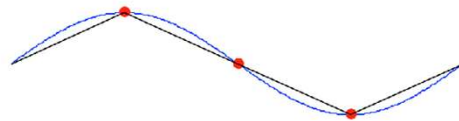
$j=1$



$$\begin{aligned} a_1 &= C \sin(\pi/4) = 0.71C \\ a_2 &= C \sin(2\pi/4) = C \\ a_3 &= C \sin(3\pi/4) = 0.71C \end{aligned}$$

$$\omega_1 = 0.59 \omega_0$$

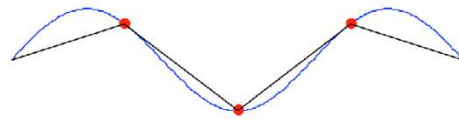
$j=2$



$$\begin{aligned} a_1 &= C \sin(2\pi/4) = C \\ a_2 &= C \sin(4\pi/4) = 0 \\ a_3 &= C \sin(6\pi/4) = -C \end{aligned}$$

$$\omega_2 = 2 \omega_0$$

$j=3$



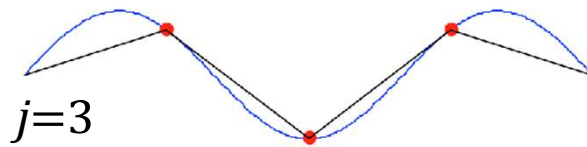
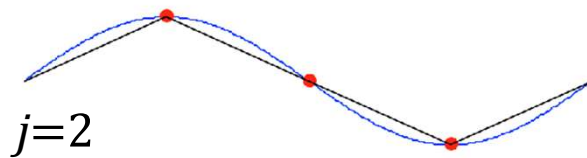
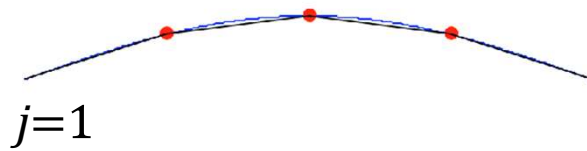
$$\begin{aligned} a_1 &= C \sin(3\pi/4) = 0.71C \\ a_2 &= C \sin(6\pi/4) = -C \\ a_3 &= C \sin(9\pi/4) = 0.71C \end{aligned}$$

$$\omega_3 = 3.4 \omega_0$$

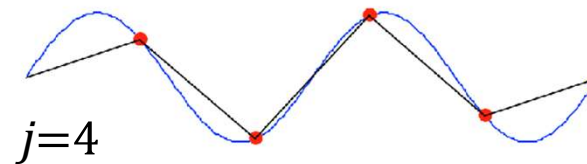
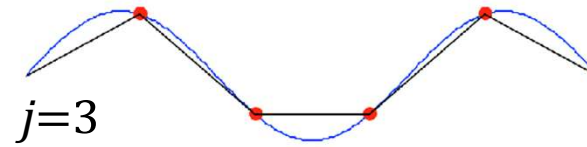
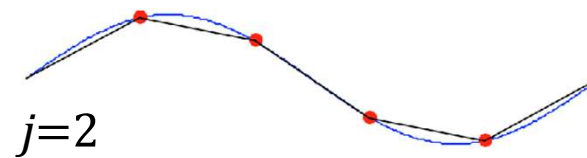
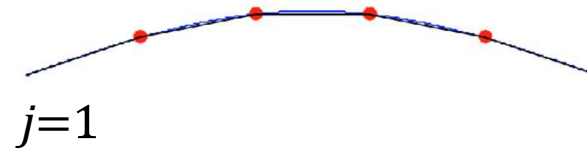


# No really, they do!

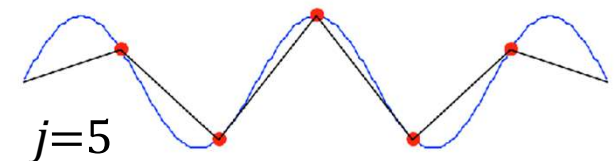
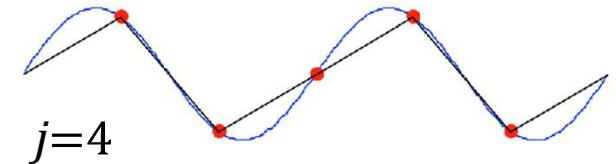
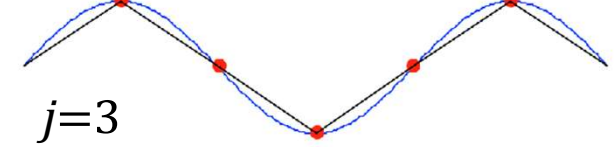
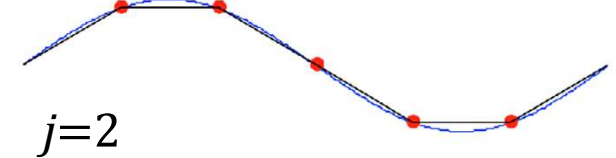
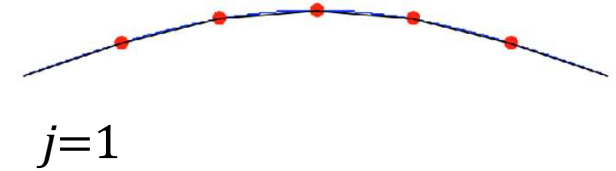
$n=3$



$n=4$

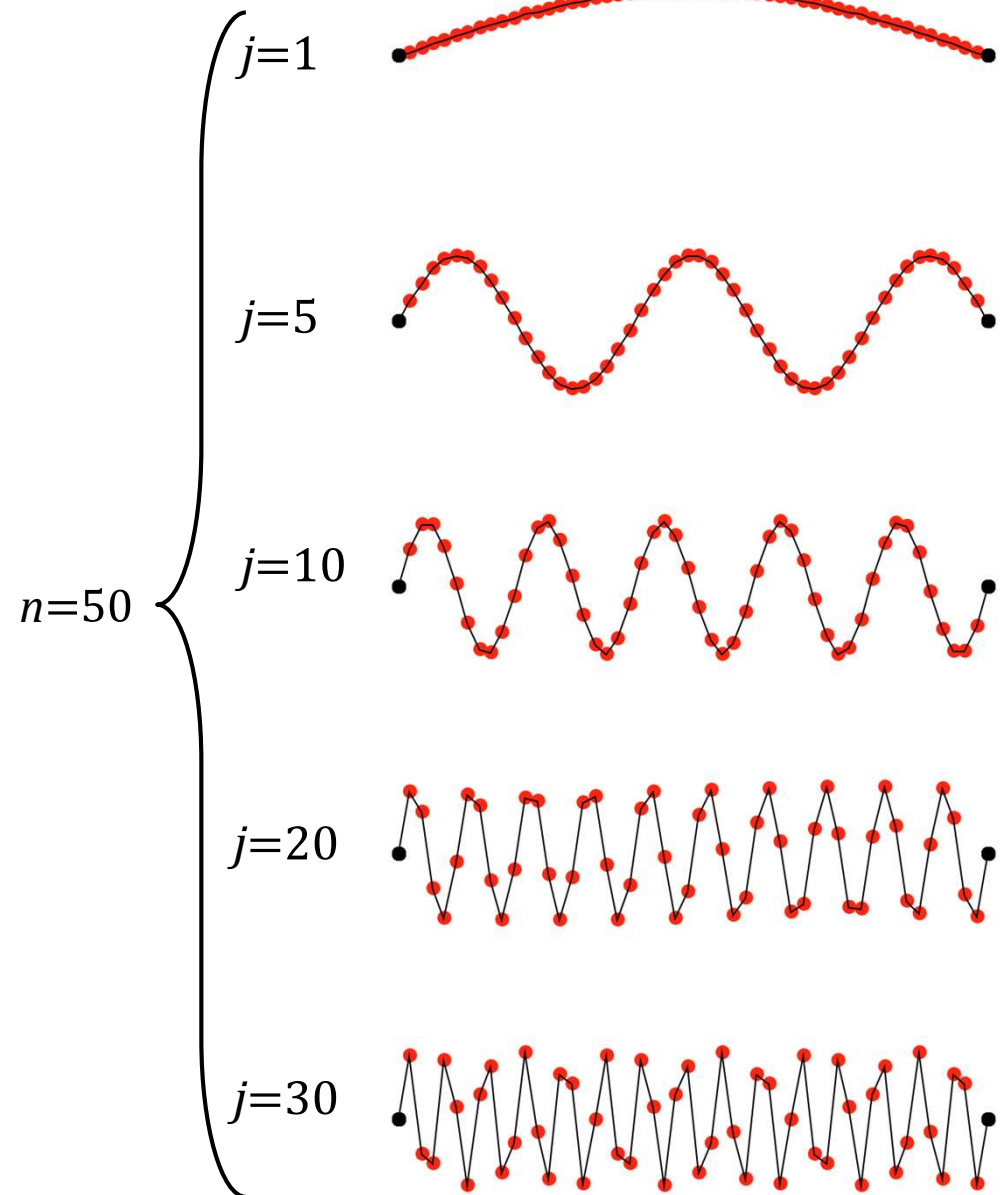
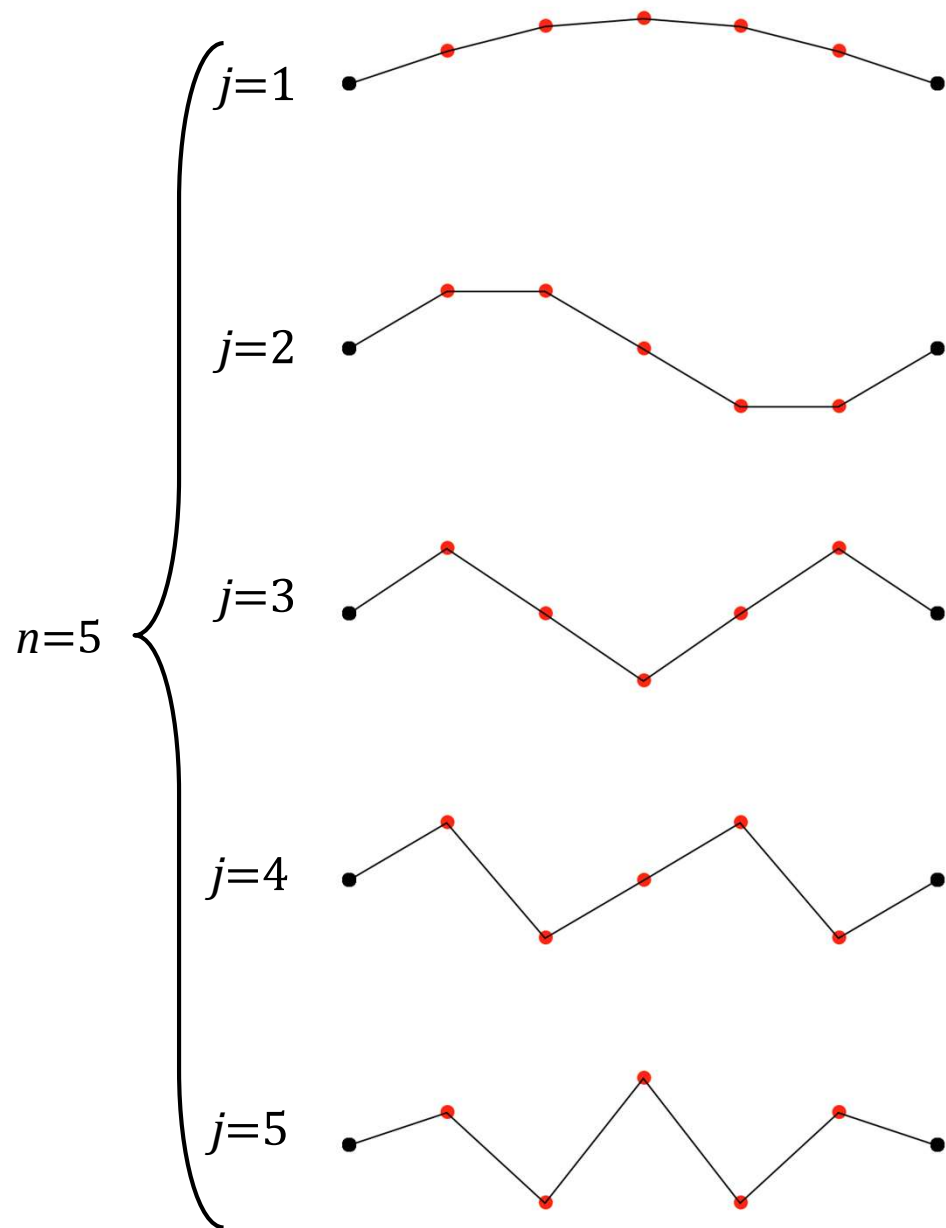


$n=5$

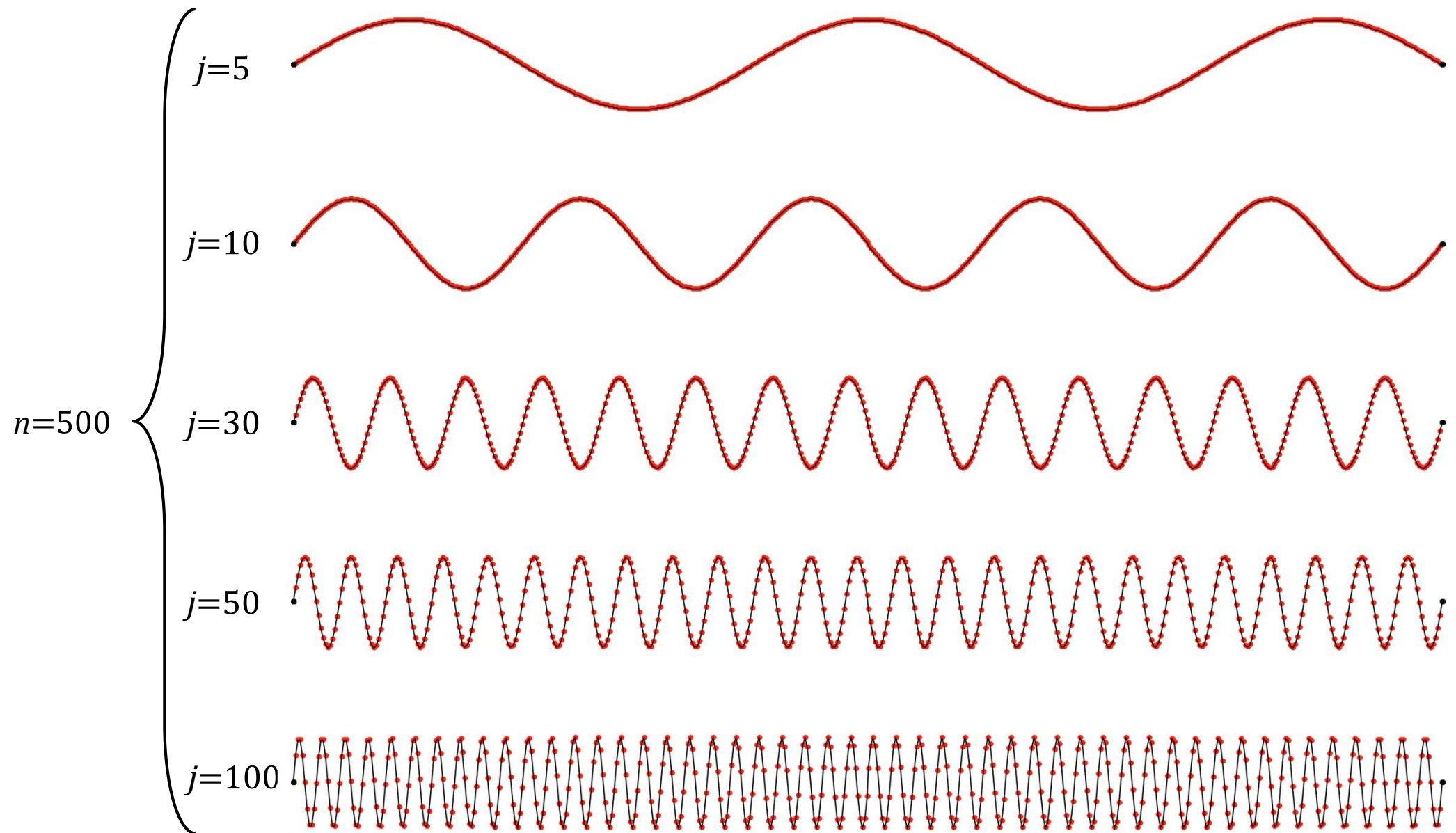


Interactive demo – <http://www.falstad.com/loadestring/>

# $n = 5$ vs $n = 50$



$n = 500$



# See? They do!

$n = \text{lots}$

$j=1$



$$\omega_1 \ll \omega_0$$

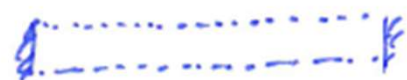
$j=2$



$j=5$



$j=n$



$$\omega_n \sim 4\omega_0$$

$$\omega_0^2 = T/ma$$

# Executive summary – loaded string

1. Equation of motion for mass  $r$  :

$$\ddot{y}_r + T/(ma) \cdot (2y_r - y_{r+1} - y_{r-1}) = 0$$

restoring force depends on local shape of the string

2. Seek normal modes with  $y_r = A_r e^{i\omega t}$

$$\Rightarrow -A_{r-1} + \left(2 - \frac{\omega^2 ma}{T}\right) A_r - A_{r+1} = 0$$

local shape of a mode is set by frequency  $\omega$

3. Adopt a sinusoidal mode shape  $A_r = C e^{i(r\theta + \phi)}$

$\theta$  sets the wavelength of the mode

$$\Rightarrow 2 \cos \theta = \frac{2\omega_0^2 - \omega^2}{\omega_0^2}$$

$\theta$  depends on  $\omega$

4. Boundary conditions determine the allowed wavelengths

$$\omega_0^2 = T/ma$$

$$\Rightarrow \theta = j\pi/(n+1), \quad j = 1, 2, \dots, n$$

$$\therefore a_r = C \sin\left(\frac{j\pi r}{n+1}\right) \quad \text{and} \quad \omega_j^2 = 2\omega_0^2 \left[1 - \cos\left(\frac{j\pi}{n+1}\right)\right]$$

mode shapes and frequencies

# Waves on a string

- This looks like standing waves on a string stretched between two fixed boundaries
  - mass is distributed continuously along the string, with linear density  $\rho$  (kg/m)
- Just for laughs, we derive the equation of motion for a “heavy” string by approximating it as  $n$  masses on a light string
  - Set  $m = \rho L/n$  and  $a = L/(n+1)$  where  $L$  is the string’s length
  - Let  $n \rightarrow \infty$ . Hilarious!
- Then we’ll derive the equation of motion using a direct approach
- Either way we end up with the *wave equation*

