



COLLEGE OF ENGINEERING  
SCHOOL OF AEROSPACE ENGINEERING

AE 6705: INTRODUCTION TO MECHATRONICS

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## LAB10

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November 28, 2021

## Table of Contents

<b>1</b>	<b>Question 1</b>	<b>2</b>
<b>2</b>	<b>Question 2</b>	<b>7</b>
<b>3</b>	<b>Question 3</b>	<b>11</b>
<b>4</b>	<b>Question 4</b>	<b>14</b>
<b>5</b>	<b>Appendix</b>	<b>15</b>
5.1	MATLAB Code . . . . .	15

## Question 1

Pick two motor velocities,  $\omega_1$  and  $\omega_2$ , greater than 90 rpm. Use your closed loop Simulink model to simulate the step response for each velocity. On each plot highlight the rise time and overshoot to show that they meet the specified requirements (you might want to look at Matlab's `stepinfo()` function for this). State the PID gains you used for these simulations.

---

### Solution:

The motor velocities chosen are

$$\omega_1 = 95 \text{ RPM} \quad \omega_2 = 120 \text{ RPM} .$$

The PID gains for the following model are

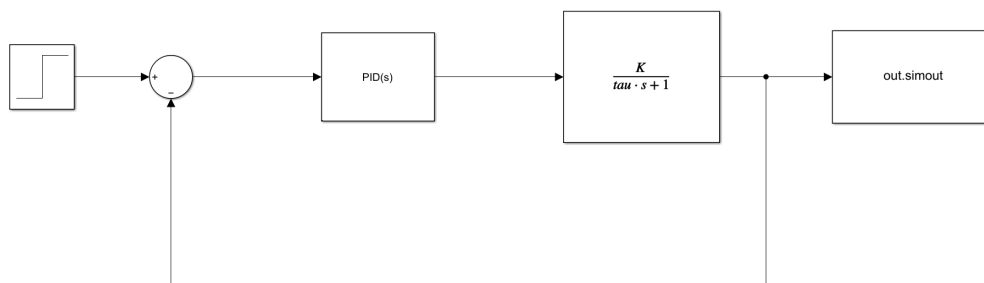


Figure 1: PID controlled feedback model

$K_P$	$K_I$	$K_D$
-0.00122960025714389	0.0382450209614726	0.000132060829813156

Table 1: PID gains tuned in Simulink

The plotted response as well as the characteristics are shown below

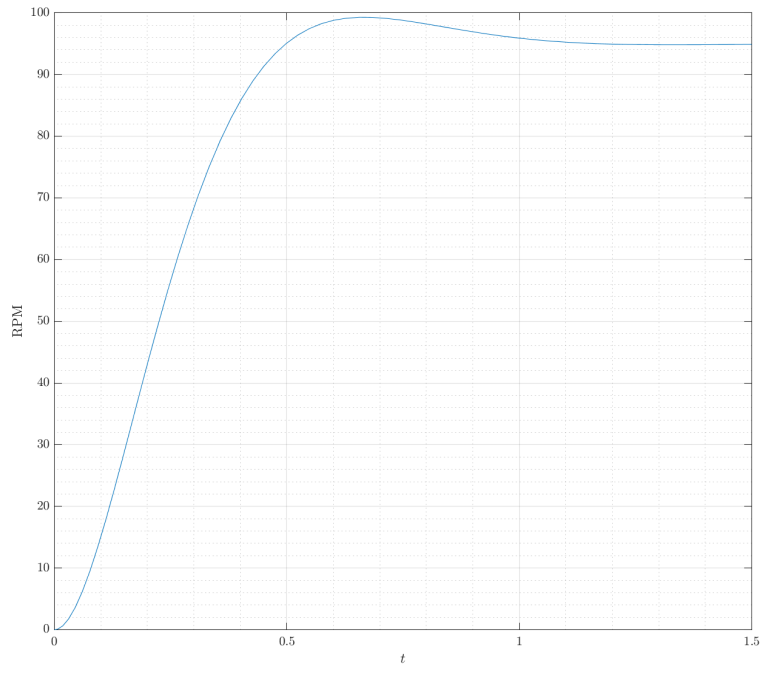


Figure 2: Step response for Simulink model for  $\omega_1 = 95$

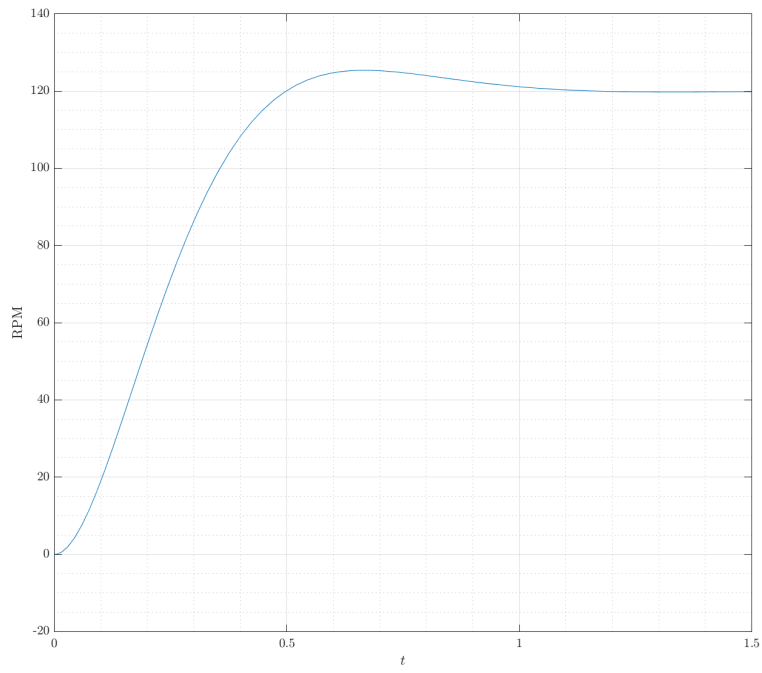


Figure 3: Step response for Simulink model for  $\omega_2 = 120$

omega1 = 95 RPM

```

    RiseTime: 0.3216
  TransientTime: 0.9124
  SettlingTime: 0.9124
  SettlingMin: 86.1496
  SettlingMax: 99.2648
    Overshoot: 4.6256
  Undershoot: 1.5809e-40
      Peak: 99.2648
    PeakTime: 0.6559

```

omega2 = 120 RPM

```

    RiseTime: 0.3214
  TransientTime: 0.9123
  SettlingTime: 0.9123
  SettlingMin: 108.1769
  SettlingMax: 125.3781
    Overshoot: 4.6181
  Undershoot: 1.5809e-40
      Peak: 125.3781
    PeakTime: 0.6800

```

---

We can confirm that both satisfy the requirement of under 5% overshoot and 500 ms rise time.

It is important to note that these results were produced by using the MATLAB [Control System Tuner](#). Where we define a tuning goal for step response tracking with the following performance requirements

$$\text{Max Overshoot} := MOS = 4.5\% \quad \longrightarrow \quad \zeta = \frac{-\ln(MOS/100)}{\sqrt{\pi^2 + \ln^2(MOS/100)}} = 0.7025$$

$$\text{if rise time} := t_r = 0.35$$

$$\therefore t_r \omega_n = \frac{1}{\sqrt{1 - \zeta^2}} \left[ \pi - \arctan \left( \frac{\sqrt{1 - \zeta^2}}{\zeta} \right) \right] \quad \text{and} \quad \text{time constant} := \tau \approx \frac{1}{\zeta \omega_n}$$

$$\therefore \tau = 0.1509.$$

Then we plug these into the settings for the tuning goals like in the following Figure. Then the app will optimize the PID gains.

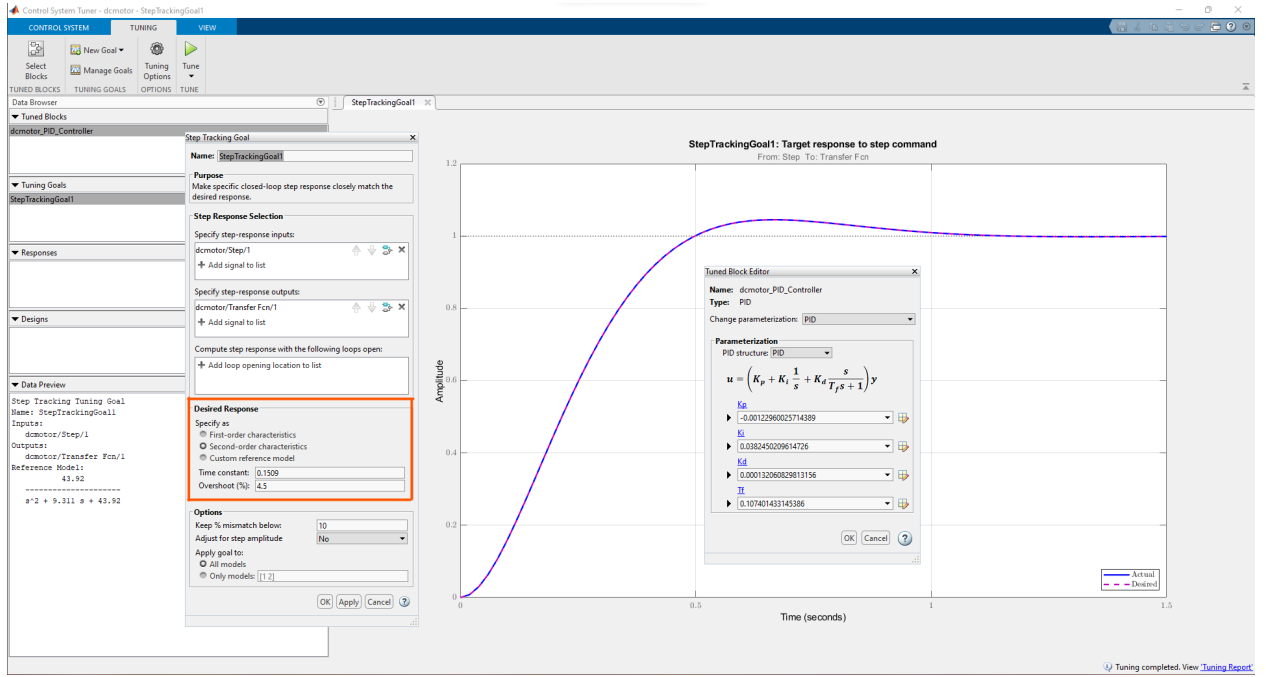


Figure 4: Tuning process with Control System Tuner

Also, we must keep in mind that the PID controller of the Simulink is actually in the form of

$$K_p + \frac{K_i}{s} + K_d \frac{Ns}{s + N}$$

where  $N$  is the filter coefficient with a value of

$$N = 9.31086272048467.$$

To implement this in MSP432, we code a high pass filter for the derivative control like in the code of Listing 1. Although, this is in the code, it is not implemented and the first order finite differencing is used instead.

```

1  /*
2  * Function: derivative_filter
3  * -----
4  * Compute output from derivative filter/high pass filter for
5  * derivative control.
6  *
7  * Arguments:
8  *   [float] de1: previous error_dot value
9  *   [float] de2: current error_dot value
10 *   [float] pre_out: previous output value from the filter
11 * Returns:
12 *   [float] filtered output value
13 */
14 float derivative_filter(float de1, float de2, float pre_out){
15     float alpha = (1/N) / ((1/N) + TIME_STEP);
16     float yout = alpha * pre_out + alpha * (de2 - de1)*Kd*N;
17     return yout;
18 }

```

Listing 1: Derivative filter c code implementation

## Question 2

Now using your *hardware* setup and the same PID gains used in simulation, control the DC motor to each speed of  $\omega_1$  and  $\omega_2$  starting from zero speed each time. Record the step response for each case. Plot the experimental step response (two plots, one for each case) and highlight the overshoot and the rise time. Does it match the required performance specifications? Now create two new plots where you overlay the simulated and experimental response. How similar is the experimental response to the simulated one? If the simulated and experimental responses look noticeably different, explain why you think that is the case.

---

### Solution:

For  $\omega_1 = 95$

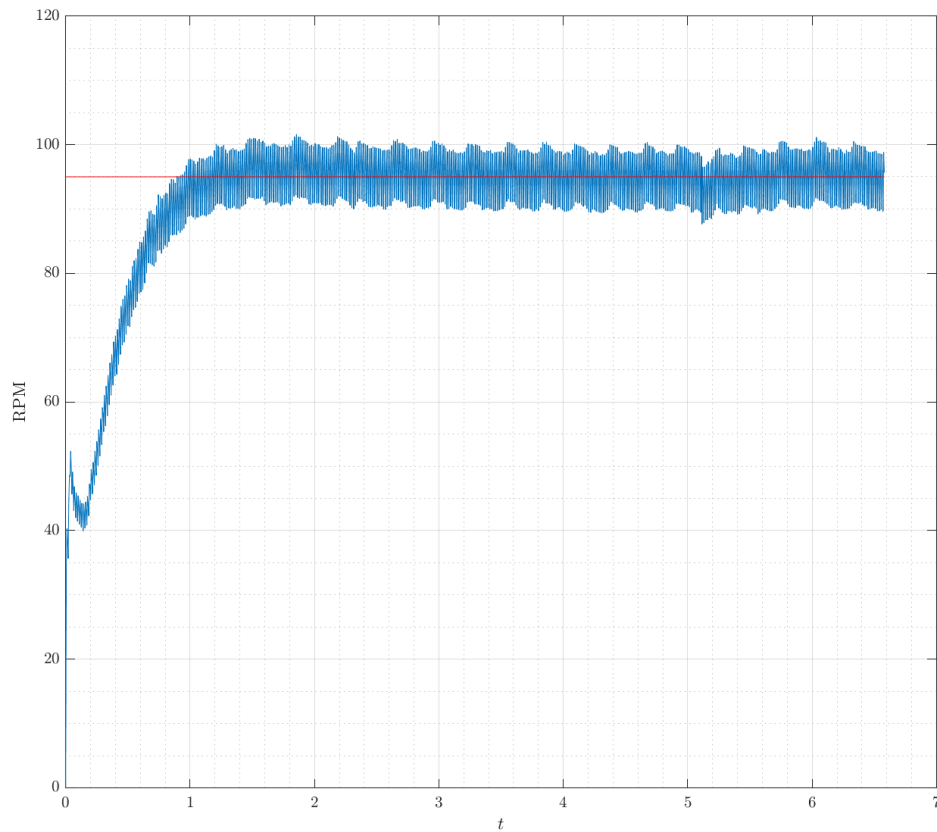


Figure 5: Actual response at  $\omega = 95$  RPM with Simulink PID gains



For  $\omega_2 = 120$

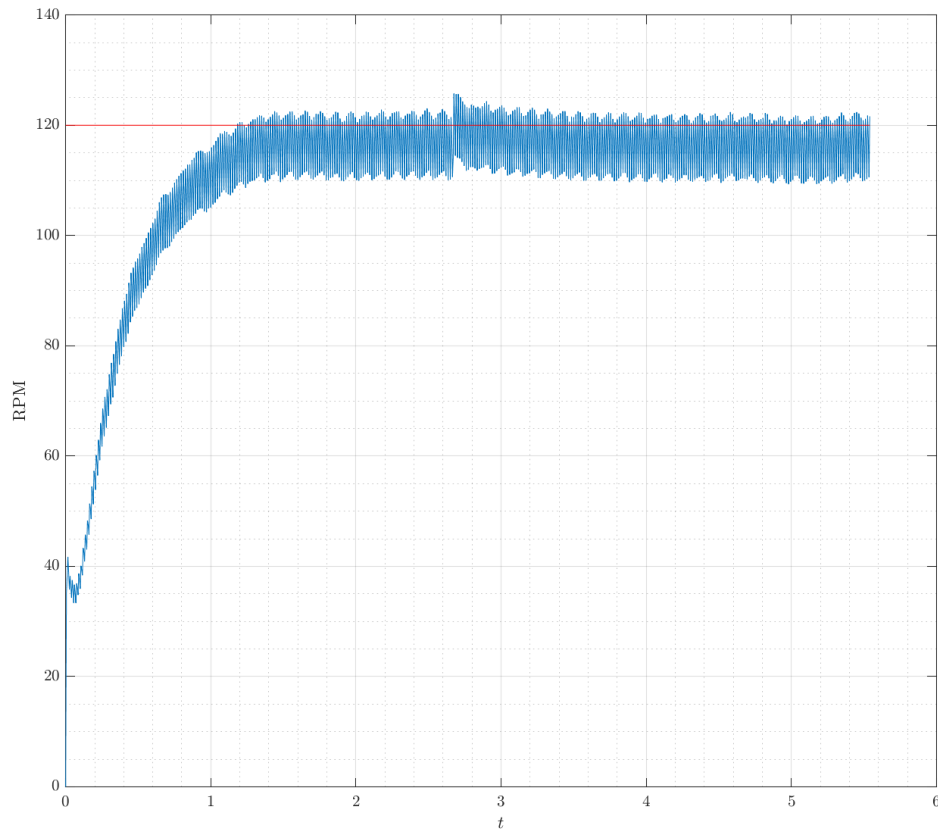


Figure 6: Actual response at  $\omega = 120$  RPM with Simulink PID gains

From below we can see that the rise time and overshoot for  $\omega_1 = 95$  RPM are not satisfied but for  $\omega_2 = 120$  the overshoot is below 5% and the rise time is above 500 ms.

---

stepinfo data
---------------

---

`omega1 = 95 RPM`

```
RiseTime: 0.6383
TransientTime: 6.5770
SettlingTime: 6.5770
SettlingMin: 80.0777
SettlingMax: 101.5857
Overshoot: 6.2170
Undershoot: 0
Peak: 101.5857
PeakTime: 1.8550
```

$\omega_2 = 120$  RPM

```
RiseTime: 0.7482
TransientTime: 5.5389
SettlingTime: 5.5389
SettlingMin: 100.3019
SettlingMax: 125.7242
Overshoot: 3.3694
Undershoot: 0
Peak: 125.7242
PeakTime: 2.6750
```

---

If we overlay the plots from problem 1 and problem 2 we have

For  $\omega_1 = 95$

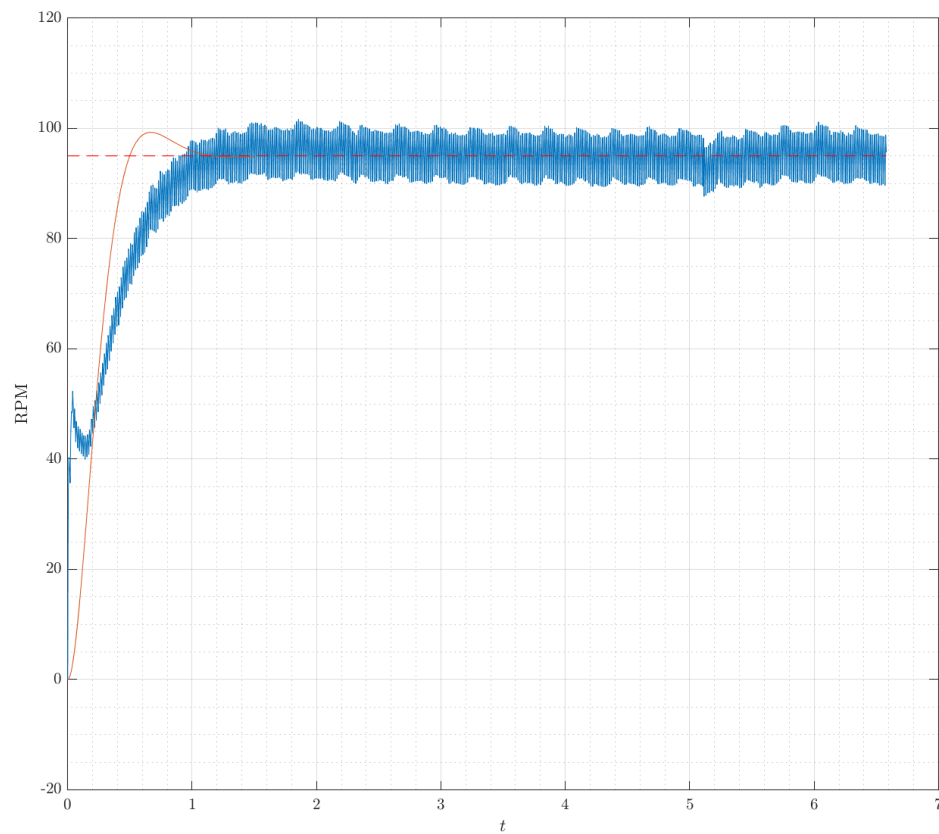


Figure 7: Actual response and simulation response at  $\omega = 95$  RPM with Simulink PID gains

For  $\omega_2 = 120$

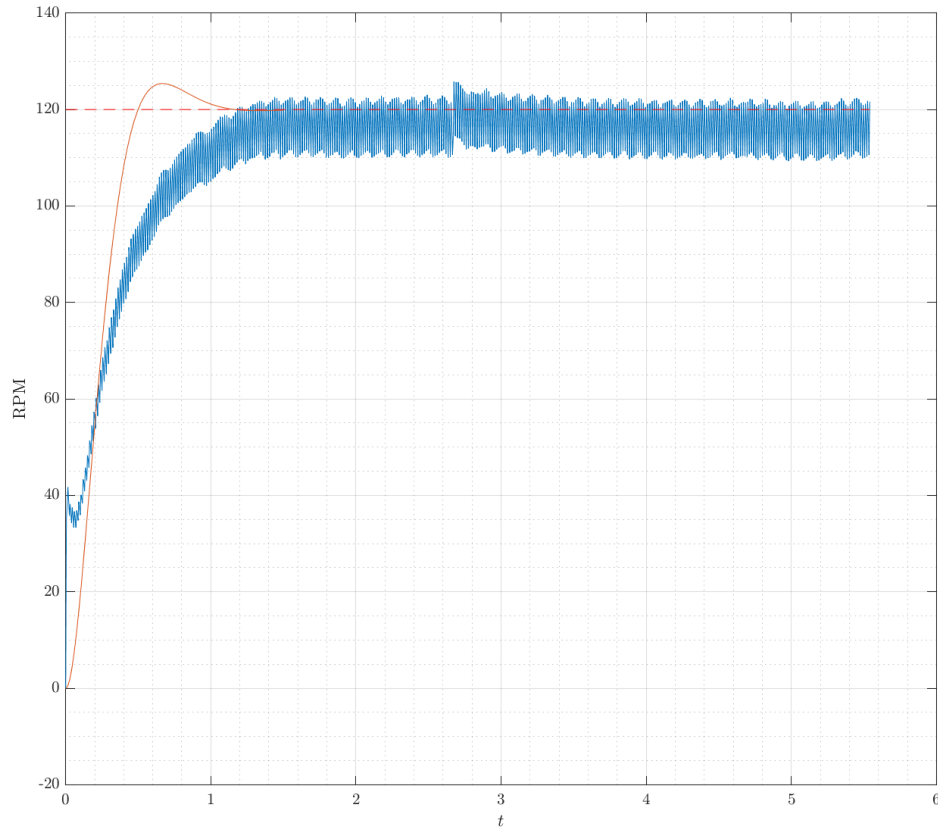


Figure 8: Actual response and simulation response at  $\omega = 120$  RPM with Simulink PID gains

Observing these plots we can see that the response seem somewhat close but the overshoot that occurs for a second order system seems to be less obvious for the actual motor response compared to the simulation response. For the higher RPM value we can see that there seems to be a bit of bias at the steady state response and for the both there is a significant amount of noise which could be caused by the amplification of error by the derivative and proportional terms. Additionally, we see a small hump at the beginning of the response for the actual motor.

### Question 3

If your experimental response does not meet the required rise time and overshoot specifications, tune your PID gains until it does. You might want to start by setting all gains to 0 and tuning your integral gain first, then your proportional gain, and finally your derivative gain. Note, the final gains might differ significantly from your simulation gains.

---

#### Solution:

We employ the following PID gains and redo the analysis

$K_P$	$K_I$	$K_D$
0.00572311362145534	0.120364997080663	-0.000158546538651373

Table 2: New PID gains tuned in Simulink

This produces the following results

---

stepinfo data

---

omega1 = 95 RPM

```
RiseTime: 0.1256
TransientTime: 1.8351
SettlingTime: 1.5603
SettlingMin: 86.4440
SettlingMax: 99.4873
Overshoot: 4.8281
Undershoot: 0
Peak: 99.4873
PeakTime: 0.4300
```

omega2 = 120 RPM

```
RiseTime: 0.3072
TransientTime: 1.6532
SettlingTime: 1.0519
SettlingMin: 104.0249
SettlingMax: 115.6861
Overshoot: 0.0299
Undershoot: 0
Peak: 115.6861
PeakTime: 1.9600
```

For  $\omega_1 = 95$

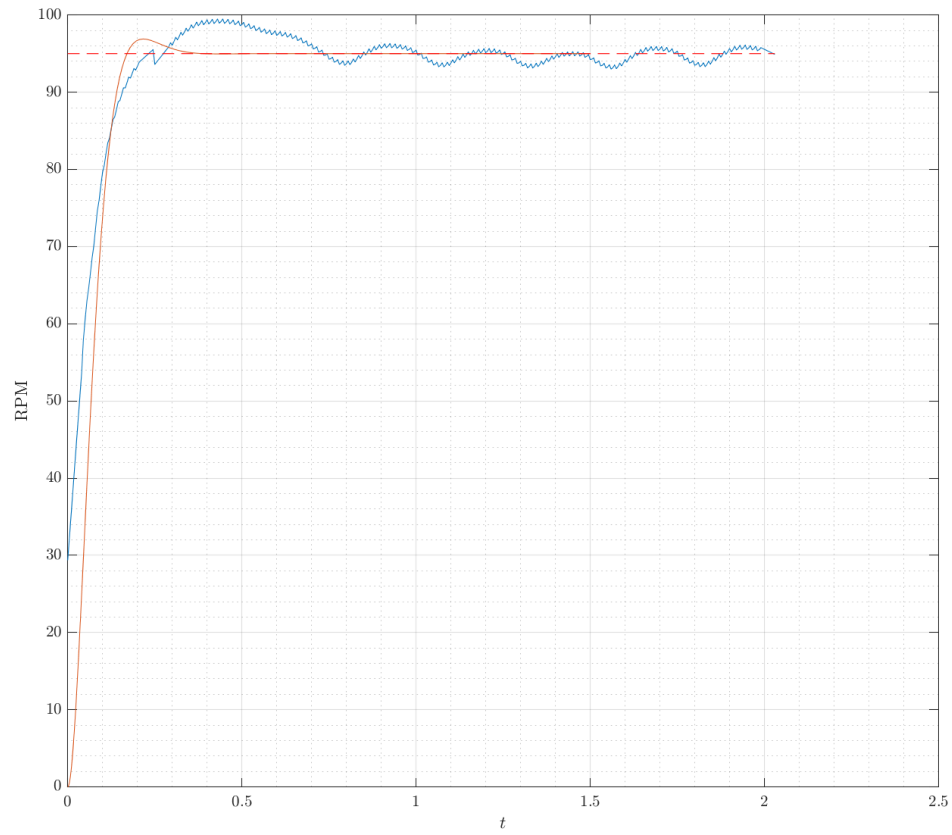


Figure 9: Actual response and simulation response at  $\omega = 95$  RPM with new Simulink PID gains

For  $\omega_2 = 120$

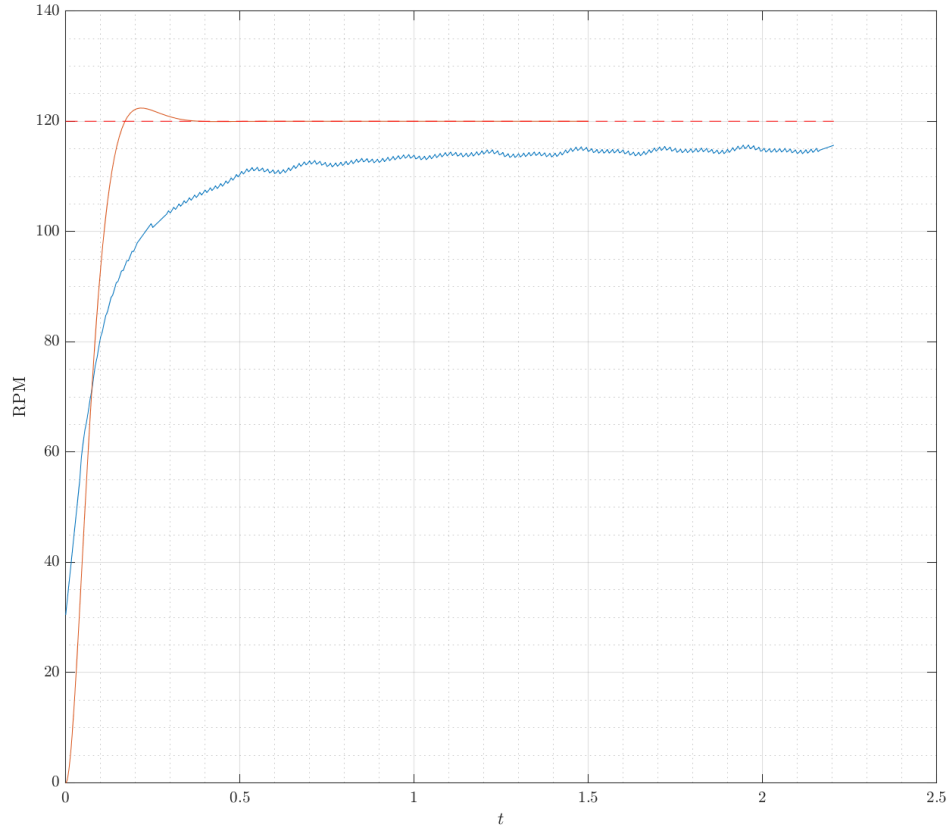


Figure 10: Actual response and simulation response at  $\omega = 120$  RPM with new Simulink PID gains

The results show that we have satisfied the requirements. These results were produced by implementing the Savitzky-Golay filtering with `sgolayfilt()` in MATLAB. But still we have a bias for the  $\omega = 120$  RPM. This means that if we are to further improve the response of this motor we will have to employ a noise filter and calibrate the RPM for high values.

## Question 4

Use the controller gains from Question 3 to control the motor speed to a value lower than 80 rpm. What do you notice in the step response? Explain the reason behind the behavior observed in your response.

---

### Solution:

With  $\omega = 60$  we have

$$t_r = 0.0391$$

$$MOS = 45.6143$$

and the overshoot is very high. In the previous lab we have seen that motor plant itself can be computed to be different with different  $K$  value and a different time constant when simulating with different duty cycles. From this we can deduce that with a plant computed for a duty cycle of 100% might deviate too much from the actual plant for when the duty cycles are low. That is the gain  $K$  value might be too high and that would amplify the response and the noise too much and result in a very high overshoot.

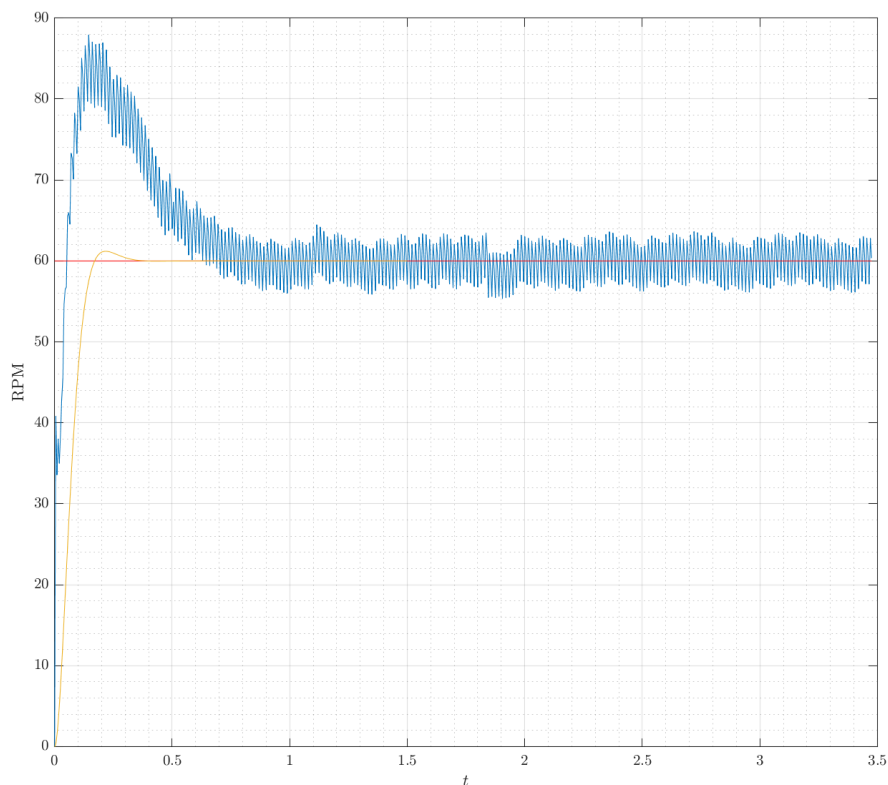


Figure 11: Overlaid results for when  $\omega = 60$

## Appendix

### 5.1 MATLAB Code

```
1 % AE6705 Lab10 MATLAB code
2 % Tomoki Koike
3 clear all; close all; clc; % housekeeping commands
4 set(groot, 'defaulttextinterpreter','latex');
5 set(groot, 'defaultAxesTickLabelInterpreter','latex');
6 set(groot, 'defaultLegendInterpreter','latex');
7 %%
8 % Gain and time constant from lab9
9 K = 1.233265815820895e+02;
10 tau = 0.075250836120401;
11 %%
12 % Max overshoot less than 5% → 4.5%
13 MOS = 4.5;
14 zeta = -log(MOS/100)/sqrt(pi^2 + log(MOS/100)^2); % 0.702504592260657
15 % Rise time less than 0.5 seconds → 0.35 seconds
16 tr = 0.35;
17 temp = 1/sqrt(1-zeta^2) * (pi - atan(sqrt(1-zeta^2)/zeta));
18 time_const = tr / zeta / temp;
19
20 % Tuned PID gains
21 % p2
22 % Kp = -0.00122960025714389;
23 % Ki = 0.0382450209614726;
24 % Kd = 0.000132060829813156;
25 % N = 9.31086272048467;
26 %
27 % p3
28 Kp = 0.00572311362145534;
29 Ki = 0.120364997080663;
30 Kd = -0.000158546538651373;
31 N = 36.097373491324220;
32 %%
33 % Simulate Simulink Model
34 omega1 = 95; % first candidate angular velocity
35 omega2 = 120; % second candidate angular velocity
36 i = 1;
37 for omega = [omega1 omega2]
38     res = sim("dcmotor.slx");
39     % Plotting
40     fig = figure("Renderer","painters","Position",[60 60 950 800]);
```



```

41     plot(res.tout, res.simout.signals.values);
42     grid on; grid minor; box on;
43     xlabel('$t$')
44     ylabel('RPM')
45     if omega == omega1
46         saveas(fig, 'outputs/model_response_p3_w=95.png');
47     elseif omega == omega2
48         saveas(fig, 'outputs/model_response_p3_w=120.png');
49     end
50     output(i) = res;
51     i = i + 1;
52 end
53 %%
54 % Response characteristics — transient and steady state
55 rc = stepinfo(res.simout.signals.values, res.tout)
56 %%
57 % omega = 95
58 % read data
59 data1 = csvread("p3_omega=95.csv");
60 data11 = data1(1:50);
61 data12 = data1(51:end);
62 % Filter the data a bit removing outliers
63 data11 = sgolayfilt(data11, 1, 17);
64 data12 = rmoutliers(data12);
65 data12 = sgolayfilt(data12, 1, 17);
66 data1 = [data11; data12];
67
68 tspan1 = 0:0.005:0.005*(length(data1)-1);
69 % Plotting
70 fig = figure("Renderer", "painters", "Position", [60 60 950 800]);
71 plot(tspan1, data1);
72 grid on; grid minor; box on; hold on;
73 plot(tspan1, 95 * ones(1, length(data1)), 'r')
74 hold off;
75 xlabel('$t$')
76 ylabel('RPM')
77 saveas(fig, 'outputs/actual_response_p3_w=95.png');
78 %%
79 % Response analysis
80 rc95 = stepinfo(data1, tspan1)
81 %%
82 % omega = 120
83 % read data
84 data2 = csvread("p3_omega=120.csv");
85 data21 = data2(1:50);

```

```

86 data22 = data2(51:end);
87 % Filter the data a bit removing outliers
88 data21 = sgolayfilt(data21,1,17);
89 data22 = rmoutliers(data22);
90 data22 = sgolayfilt(data22,1,17);
91 data2 = [data21; data22];
92
93 tspan2 = 0:0.005:0.005*(length(data2)-1);
94 % Plotting
95 fig = figure("Renderer","painters","Position",[60 60 950 800]);
96 plot(tspan2, data2);
97 grid on; grid minor; box on; hold on;
98 plot(tspan2, 120 * ones(1, length(data2)), 'r')
99 hold off;
100 xlabel('$t$')
101 ylabel('RPM')
102 saveas(fig, 'outputs/actual_response_p3_w=120.png');
103 %%
104 % Response analysis
105 rc120 = stepinfo(data2, tspan2)
106 %%
107 % Overlaid plots
108 res = output(1);
109 fig = figure("Renderer","painters","Position",[60 60 950 800]);
110 plot(tspan1, data1);
111 grid on; grid minor; box on; hold on;
112 plot(res.tout, res.simout.signals.values)
113 plot(tspan1, 95 * ones(1, length(data1)), '—r')
114 hold off;
115 xlabel('$t$')
116 ylabel('RPM')
117 saveas(fig, 'outputs/actual_response_overlaid_p3_w=95.png');
118 %%
119 res = output(2);
120 fig = figure("Renderer","painters","Position",[60 60 950 800]);
121 plot(tspan2, data2);
122 grid on; grid minor; box on; hold on;
123 plot(res.tout, res.simout.signals.values)
124 plot(tspan2, 120 * ones(1, length(data2)), '—r')
125 hold off;
126 xlabel('$t$')
127 ylabel('RPM')
128 saveas(fig, 'outputs/actual_response_overlaid_p3_w=120.png');
129 %%
130 % P4: omega = 60

```

```

131 % read data
132 data4 = csvread("p4_omega=60.csv");
133 data41 = data4(1:100);
134 data42 = data4(101:end);
135 % Filter the data a bit removing outliers
136 % data41 = sgolayfilt(data41,1,17);
137 data42 = rmoutliers(data42);
138 % data42 = sgolayfilt(data42,1,17);
139 data4 = [data41; data42];
140
141 tspan4 = 0:0.005:0.005*(length(data4)-1);
142 % Plotting
143 omega = 60;
144 res = sim("dcmotor.slx");
145
146 fig = figure("Renderer","painters","Position",[60 60 950 800]);
147 plot(tspan4, data4);
148 grid on; grid minor; box on; hold on;
149 plot(tspan4, 60 * ones(1, length(data4)), 'r')
150 plot(res.tout, res.simout.signals.values);
151 hold off;
152 xlabel('$t$')
153 ylabel('RPM')
154 saveas(fig, 'outputs/actual_response_p4_w=60.png');
155 %%
156 % Response analysis
157 rc60 = stepinfo(data4, tspan4)

```