



COLLEGE OF ENGINEERING  
SCHOOL OF AEROSPACE ENGINEERING

ME 6444: NONLINEAR SYSTEMS

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## HW4

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## Problem 1

(30 points) Discrete Nonlinear Modelling, Hamiltonian, and Phase Plane Analysis

- (a) Use Lagrange's equations to verify that the spinning shaft pictured is governed by

$$\frac{1}{4}m\ddot{\theta} - \frac{1}{4}m\dot{\theta}\tan\theta - \frac{1}{4}m\Omega^2\tan\theta + \frac{1}{2}k\frac{(1-\cos\theta)}{\cos\theta}\tan\theta = 0,$$

where  $\Omega$  is an imposed constant rotational spin. Assume mass-less links and an unstretched spring when  $\theta$  is zero. Neglect gravitational potential energy (small compared to stored elastic energy).

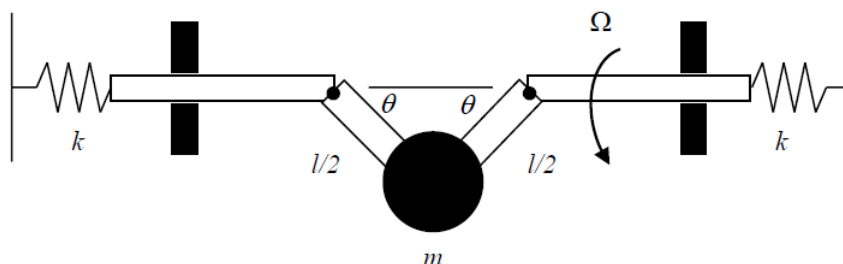


Figure 1: Spinning shaft system

- (b) Sketch the local trajectories in phase plane about each fixed point assuming  $\Omega > 0$ .
- (c) Show that the total energy  $E = T + V$  is not conserved, and instead that the Hamiltonian  $H = L - \frac{\partial L}{\partial \dot{\theta}}\dot{\theta}$  is conserved.
- (d) Using,  $H$ , determine the stability of the fixed points.

### Solution:

(a) First we derive the kinematics of this system to compute the position and velocity of the mass,  $m$ . From Figure 2 we define the reference frames, e-frame and a-frame, and the position vector of the point mass can be expressed as

$$\vec{r}_{OA} = \frac{l}{2} \sin\theta \hat{a}_1.$$

Then the velocity vector with respect to the e-frame becomes

$$\begin{aligned} {}^e\vec{v}_{OA} &= {}^e \frac{d\vec{r}_{OA}}{dt} \\ &= {}^a \frac{d}{dt} \left( \frac{l}{2} \sin\theta \hat{a}_1 \right) + \Omega \hat{e}_1 \times \frac{l}{2} \sin\theta \hat{a}_1 \\ &= \frac{l}{2} \dot{\theta} \cos\theta \hat{a}_1 + \frac{l}{2} \Omega \sin\theta \hat{a}_2. \end{aligned}$$

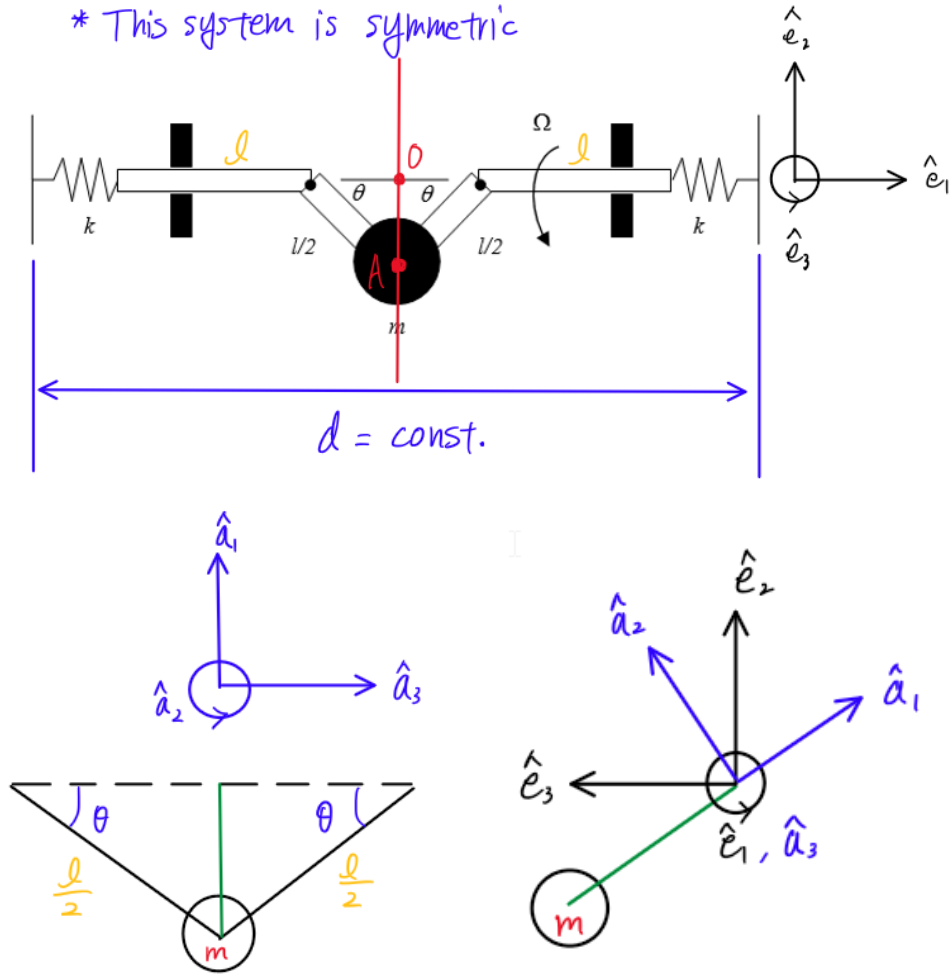


Figure 2: Sktech of diagram from front, top, and side view with reference frames

Then the kinetic energy becomes

$$\begin{aligned}
 T &= \frac{1}{2} m ({}^e \vec{v}_{OA} \cdot {}^e \vec{v}_{OA}) \\
 &= \frac{1}{8} m l^2 \dot{\theta}^2 \cos^2 \theta + \frac{1}{8} m l^2 \Omega^2 \sin^2 \theta.
 \end{aligned}$$

On the other hand, the potential energy is governed by the stored elastic energy and we can ignore the gravitational potential energy since it is small compared to the stored elastic energy.

$$\begin{aligned}
 V &= \frac{1}{2} k \left( \frac{l}{2} (1 - \cos \theta) \right)^2 = \frac{1}{2} k \left( \frac{l}{2} (1 - \cos \theta) \right)^2 \\
 &= \frac{1}{4} k l^2 (1 - \cos \theta)^2
 \end{aligned}$$

Since the Lagrangian is  $L = T - V$  we can compute

$$\begin{aligned}\frac{\partial L}{\partial \dot{\theta}} &= \frac{1}{4}ml^2\dot{\theta}\cos^2\theta \\ \frac{\partial}{\partial t}\left(\frac{\partial L}{\partial \dot{\theta}}\right) &= \frac{1}{4}ml^2\ddot{\theta}\cos^2\theta - \frac{1}{2}ml^2\dot{\theta}^2\cos\theta\sin\theta \\ \frac{\partial L}{\partial \theta} &= -\frac{1}{4}ml^2\dot{\theta}^2\cos\theta\sin\theta + \frac{1}{4}ml^2\Omega^2\sin\theta\cos\theta - \frac{1}{2}kl^2(1 - \cos\theta)\sin\theta.\end{aligned}$$

Thus from

$$\frac{\partial}{\partial t}\left(\frac{\partial L}{\partial \dot{\theta}}\right) - \frac{\partial L}{\partial \theta} = 0$$

we have

$$\begin{aligned}\frac{1}{4}ml^2\ddot{\theta}\cos^2\theta - \frac{1}{2}ml^2\dot{\theta}^2\cos\theta\sin\theta + \frac{1}{4}ml^2\dot{\theta}^2\cos\theta\sin\theta - \frac{1}{4}ml^2\Omega^2\sin\theta\cos\theta + \frac{1}{2}kl^2(1 - \cos\theta)\sin\theta &= 0 \\ \frac{1}{4}ml^2\ddot{\theta}\cos^2\theta - \frac{1}{4}ml^2\dot{\theta}^2\cos\theta\sin\theta - \frac{1}{4}ml^2\Omega^2\sin\theta\cos\theta + \frac{1}{2}kl^2(1 - \cos\theta)\sin\theta &= 0\end{aligned}$$

Hence,

$$\frac{1}{4}m\ddot{\theta} - \frac{1}{4}m\dot{\theta}^2\tan\theta - \frac{1}{4}m\Omega^2\tan\theta + \frac{1}{2}k\frac{(1 - \cos\theta)}{\cos\theta}\tan\theta = 0.$$

(b) If we rewrite the given EOM with  $x_1 = \theta$  and  $x_2 = \dot{\theta}$  we have

$$\begin{aligned}\dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_2^2\tan x_1 + \Omega^2\tan x_1 - \frac{2k(1 - \cos x_1)}{m\cos x_1}\tan x_1\end{aligned}$$

Then the fixed point or equilibrium points can be found by solving for  $\dot{x}_1 = 0$  and  $\dot{x}_2 = 0$ .

$$\begin{aligned}x_2 = 0 &\longrightarrow x_{2e} = 0 \\ 0 &= \Omega^2\tan x_1 - \frac{2k(1 - \cos x_1)}{m\cos x_1}\tan x_1 \\ 0 &= \tan x - 1\left(\Omega^2 - \frac{2k(1 - \cos x_1)}{m\cos x_1}\right).\end{aligned}$$

Which gives,

$$x_{1e} = n\pi \quad n = 0, 1, 2, \dots$$

or

$$x_{1e} = \arccos\left(\frac{2k}{2k + m\Omega^2}\right)$$

and

$$x_{2e} = 0.$$

If we declare the system matrix  $A$  to be

$$A = \begin{bmatrix} a & b \\ c & d \end{bmatrix}$$

we know that

$$a = 0,$$

The elements  $a, b, c, d$  are

$$\begin{aligned} a &= \frac{\partial \dot{x}_1}{\partial x_1} = 0 \\ b &= \frac{\partial \dot{x}_1}{\partial x_2} = 1 \\ c &= \frac{\partial \dot{x}_2}{\partial x_1} = x_2^2 \sec^2 x_1 + \Omega^2 \sec^2 x_1 - \frac{2k}{m} \sin x_1 \sec x_1 \tan x_1 \\ &\quad - \frac{2k}{m} (1 - \cos x_1) \sec x_1 \tan^2 x_1 - \frac{2k}{m} (1 - \cos x_1) \sec^3 x_1 \\ d &= \frac{\partial \dot{x}_2}{\partial x_2} = 2x_2 \tan x_1 \end{aligned}$$

Now if we plug in the equilibrium points into this matrix we can analyze the local trajectories in the phase plane. For the equilibrium point  $(n\pi, 0)$  we have

$$A = \begin{cases} \begin{bmatrix} 0 & 1 \\ \Omega^2 & 0 \end{bmatrix} & \text{if } n \text{ is even} \\ \begin{bmatrix} 0 & 1 \\ \frac{4k}{m} + \Omega^2 & 0 \end{bmatrix} & \text{if } n \text{ is odd} \end{cases}$$

For the case where  $n$  is [even](#) the eigenvalues and eigenvectors are

$$\lambda_1 = \Omega, \quad \lambda_2 = -\Omega, \quad v = \left\{ \begin{bmatrix} \frac{1}{\Omega} \\ 1 \end{bmatrix}, \begin{bmatrix} -\frac{1}{\Omega} \\ 1 \end{bmatrix} \right\}.$$

This shows that this equilibrium point is a saddle point.

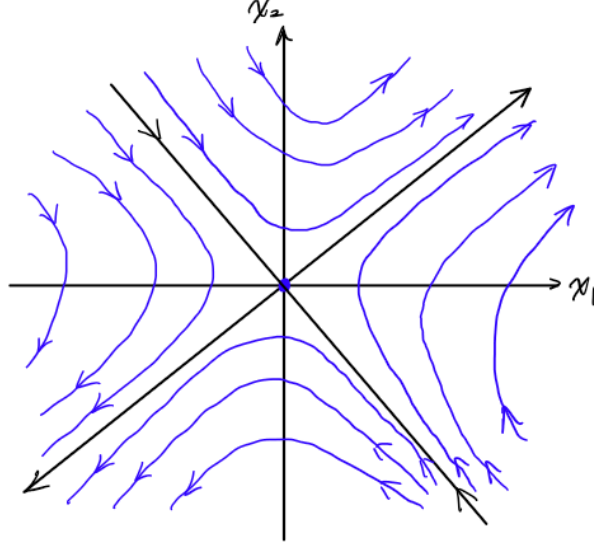


Figure 3: Saddle point equilibrium point and local trajectories

Similarly, when  $n$  is odd we have

$$\lambda_1 = \frac{\sqrt{m(m\Omega^2 + 4k)}}{m\Omega^2 + 4k}, \quad \lambda_2 = -\frac{\sqrt{m(m\Omega^2 + 4k)}}{m\Omega^2 + 4k},$$

$$v = \left\{ \begin{bmatrix} \frac{\sqrt{m(m\Omega^2 + 4k)}}{m\Omega^2 + 4k} \\ 1 \end{bmatrix}, \begin{bmatrix} -\frac{\sqrt{m(m\Omega^2 + 4k)}}{m\Omega^2 + 4k} \\ 1 \end{bmatrix} \right\}$$

This is also a saddle point, and therefore, the local trajectories of this equilibrium point is the same as Figure 3.

Next, if we plug in the equilibrium point of

$$\left( \arccos\left(\frac{2k}{2k + m\Omega^2}\right), 0 \right)$$

we have the  $A$  matrix of

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{0.2500 \Omega^2 (\Omega^4 m^2 + 6 \Omega^2 k m + 8 k^2)}{k^2} & 0 \end{bmatrix}$$

and the eigenvalues and eigenvectors become

$$\lambda_1 = -\frac{\Omega \sqrt{-(m\Omega^2 + 2k)(m\Omega^2 + 4k)}}{2k}, \quad \lambda_2 = \frac{0.5000 \Omega \sqrt{-(m\Omega^2 + 2k)(m\Omega^2 + 4k)}}{2k}$$

$$v = \left\{ \begin{bmatrix} \frac{2 \Omega k \sqrt{-(m\Omega^2 + 2k)(m\Omega^2 + 4k)}}{\Omega^6 m^2 + 6 \Omega^4 k m + 8 \Omega^2 k^2} \\ 1 \end{bmatrix}, \begin{bmatrix} -\frac{2 \Omega k \sqrt{-(m\Omega^2 + 2k)(m\Omega^2 + 4k)}}{\Omega^6 m^2 + 6 \Omega^4 k m + 8 \Omega^2 k^2} \\ 1 \end{bmatrix} \right\}$$

For this equilibrium point, we have pure imaginary eigenvalues and a geometric multiplicity of 2. Thus, it is a center which is sketched as follows.

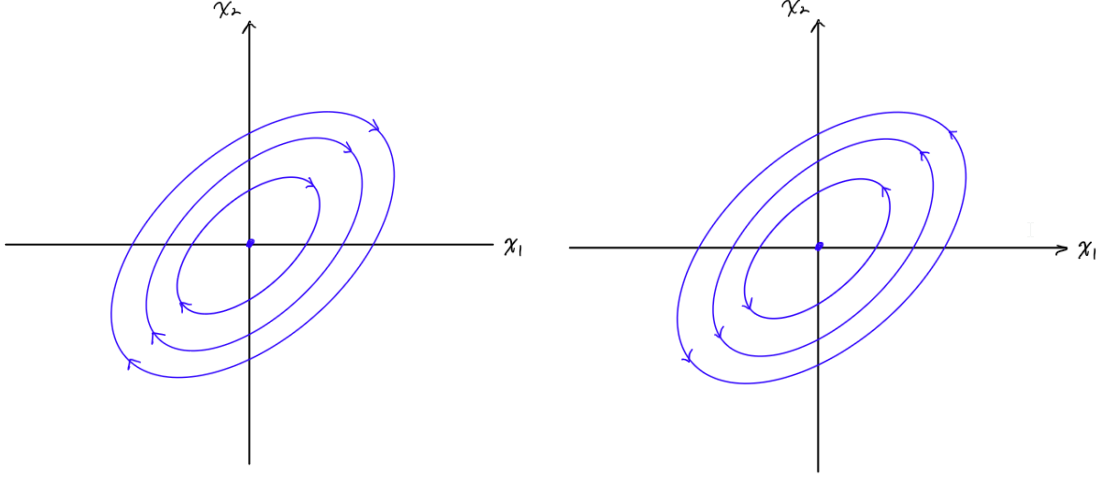


Figure 4: Center equilibrium point

Since, the direction of circulation for the center depends on the constants  $\Omega$ ,  $k$ , and  $m$  it could be circulating in either direction.

(c) We know that the Hamiltonian is

$$\begin{aligned}
 H &= L - \frac{L}{\dot{\theta}} \\
 &= \frac{1}{8}ml^2\dot{\theta}^2 \cos^2 \theta + \frac{1}{8}ml^2\Omega^2 \sin^2 \theta - \frac{1}{4}kl^2(1 - \cos \theta)^2 - \dot{\theta} \left( \frac{1}{4}ml^2\dot{\theta} \cos^2 \theta \right) \\
 &= -\frac{1}{8}ml^2\dot{\theta}^2 \cos^2 \theta + \frac{1}{8}ml^2\Omega^2 \sin^2 \theta - \frac{1}{4}kl^2(1 - \cos \theta)^2.
 \end{aligned}$$

Then, we compute

$$\begin{aligned}
 \frac{\partial H}{\partial \dot{\theta}} &= -\frac{1}{4}ml^2\dot{\theta} \cos^2 \theta \\
 \frac{\partial H}{\partial \theta} &= \frac{1}{4}ml^2\dot{\theta}^2 \cos \theta \sin \theta + \frac{1}{4}ml^2\Omega^2 \sin \theta \cos \theta - \frac{1}{2}kl^2(1 - \cos \theta) \sin \theta.
 \end{aligned}$$



Next we calculate

$$\begin{aligned}
\frac{dH}{dt} &= \frac{\partial H}{\partial \dot{\theta}} \frac{\partial \dot{\theta}}{\partial t} + \frac{\partial H}{\partial \theta} \frac{\partial \theta}{\partial t} \\
&= \left( -\frac{1}{4} m l^2 \dot{\theta} \cos^2 \theta \right) \ddot{\theta} + \left( \frac{1}{4} m l^2 \dot{\theta}^2 \cos \theta \sin \theta + \frac{1}{4} m l^2 \Omega^2 \sin \theta \cos \theta - \frac{1}{2} k l^2 (1 - \cos \theta) \sin \theta \right) \dot{\theta} \\
&= -\frac{\dot{\theta}}{l^2 \cos^2 \theta} \underbrace{\left( \frac{1}{4} m \ddot{\theta} - \frac{1}{4} m \dot{\theta}^2 \tan \theta - \frac{1}{4} m \Omega^2 \tan \theta + \frac{1}{2} k \frac{(1 - \cos \theta)}{\cos \theta} \tan \theta \right)}_{\text{EOM}=0} \\
&= 0
\end{aligned}$$

Hence, we can conclude that

$$\frac{dH}{dt} = 0.$$

This posits that the **Hamiltonian is conserved**.

Whereas, the total energy is

$$E = T + V = \frac{1}{8} m l^2 \dot{\theta}^2 \cos^2 \theta + \frac{1}{8} m l^2 \Omega^2 \sin^2 \theta + \frac{1}{4} k l^2 (1 - \cos \theta)^2.$$

This equation is not the same as the Hamiltonian, which implies that the **total energy of this system  $E$  is not conserved**.

(d) From the Hamiltonian we can find the elements of the system matrix  $A$

$$\begin{bmatrix} a & b \\ c & d \end{bmatrix}$$

where

$$\begin{aligned}
a &= \frac{1}{2} l^2 m \dot{\theta} \cos(\theta) \sin(\theta) \\
b &= -\frac{1}{4} l^2 m \cos(\theta)^2 \\
c &= \frac{1}{2} k l^2 \sin(\theta)^2 - \frac{1}{4} \Omega^2 l^2 m \cos(\theta)^2 + \frac{1}{4} \Omega^2 l^2 m \sin(\theta)^2 - \frac{1}{4} l^2 m \dot{\theta}^2 \cos(\theta)^2 \\
&\quad - \frac{1}{2} k l^2 \cos(\theta) (\cos(\theta) - 1) + \frac{1}{4} l^2 m \dot{\theta}^2 \sin(\theta)^2 \\
d &= -\frac{1}{2} l^2 m \dot{\theta} \cos(\theta) \sin(\theta)
\end{aligned}$$

and then

$$\begin{aligned}
p &= a + d = 0 \quad \text{identically zero} \\
q &= ad - bc \\
&= \frac{1}{4} l^2 m \cos(\theta)^2 \left( \frac{1}{2} k l^2 \sin(\theta)^2 - \frac{1}{4} \Omega^2 l^2 m \cos(\theta)^2 + \frac{1}{4} \Omega^2 l^2 m \sin(\theta)^2 \right. \\
&\quad \left. - \frac{1}{4} l^2 m \dot{\theta}^2 \cos(\theta)^2 - \frac{1}{2} k l^2 \cos(\theta) (\cos(\theta) - 1) + \frac{1}{4} l^2 m \dot{\theta}^2 \sin(\theta)^2 \right) \\
&\quad - \frac{1}{4} l^4 m^2 \dot{\theta}^2 \cos(\theta)^2 \sin(\theta)^2
\end{aligned}$$

For the equilibrium point  $(n\pi, 0)$ ,  $q$  becomes

$$q = \begin{cases} -\frac{1}{4} l^2 m \left( \frac{1}{4} m \Omega^2 l^2 + k l^2 \right) < 0 & \text{if } n \text{ is odd} \\ -\frac{1}{16} \Omega^2 l^4 m^2 < 0 & \text{if } n \text{ is even} \end{cases}$$

Thus, **the equilibrium point of  $(n\pi, 0)$  is an unstable saddle**. Then for the equilibrium point of  $\left( \arccos\left(\frac{2k}{2k + m\Omega^2}\right), 0 \right)$  we have

$$q = \frac{\Omega^2 k^2 l^4 m^2 (m \Omega^2 + 4k)}{4(m \Omega^2 + 2k)^3} > 0.$$

Hence, **this equilibrium point is a stable center from the Hamiltonian stability analysis**.

## Problem 2

(20 points) Phase Plane with Stability Analysis

Determine the characteristic (type of fixed point and stability) of all fixed points associated with each equation of motion below. Plot the phase plane for each.

(a)  $\ddot{u} + 2\mu\dot{u} + u + u^3 = 0$

(b)  $\ddot{u} + 2\mu\dot{u} + u - u^3 = 0$

(c)  $\ddot{u} + 2\mu\dot{u} - u + u^3 = 0$

(d)  $\ddot{u} + 2\mu\dot{u} - u - u^3 = 0$

In all cases treat the damping coefficient  $\mu$  as greater than zero.

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### Solution:

(a) Let  $x_1 = u$  and  $x_2 = \dot{u}$ , then system can be expressed in the form of

$$\begin{cases} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -2\mu x_2 - x_1 - x_1^3 \end{cases}$$

The elements  $a$ ,  $b$ ,  $c$ ,  $d$  are

$$a = \frac{\partial \dot{x}_1}{\partial x_1} = 0$$

$$b = \frac{\partial \dot{x}_1}{\partial x_2} = 1$$

$$c = \frac{\partial \dot{x}_2}{\partial x_1} = -3x_1^2 - 1$$

$$d = \frac{\partial \dot{x}_2}{\partial x_2} = -2\mu$$

which gives

$$A = \begin{bmatrix} 0 & 1 \\ -3x_1^2 - 1 & -2\mu \end{bmatrix}.$$

The equilibrium points for this system is  $x_{1e} = x_{2e} = 0$ . At this point the system matrix  $A$  becomes

$$A_e = \begin{bmatrix} 0 & 1 \\ -1 & -2\mu \end{bmatrix}.$$

and the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu + \sqrt{\mu^2 - 1}, \quad \lambda_2 = -\mu - \sqrt{\mu^2 - 1}$$

$$v_1 = \begin{bmatrix} -\mu - \sqrt{\mu^2 - 1} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} -\mu + \sqrt{\mu^2 - 1} \\ 1 \end{bmatrix}$$

Thus, the equilibrium points are

for  $(0, 0)$

→ if  $\mu > 1$ : a stable node

→ if  $\mu = 1$ : a stable singular node

→ if  $0 < \mu < 1$ : a stable spiral/focus

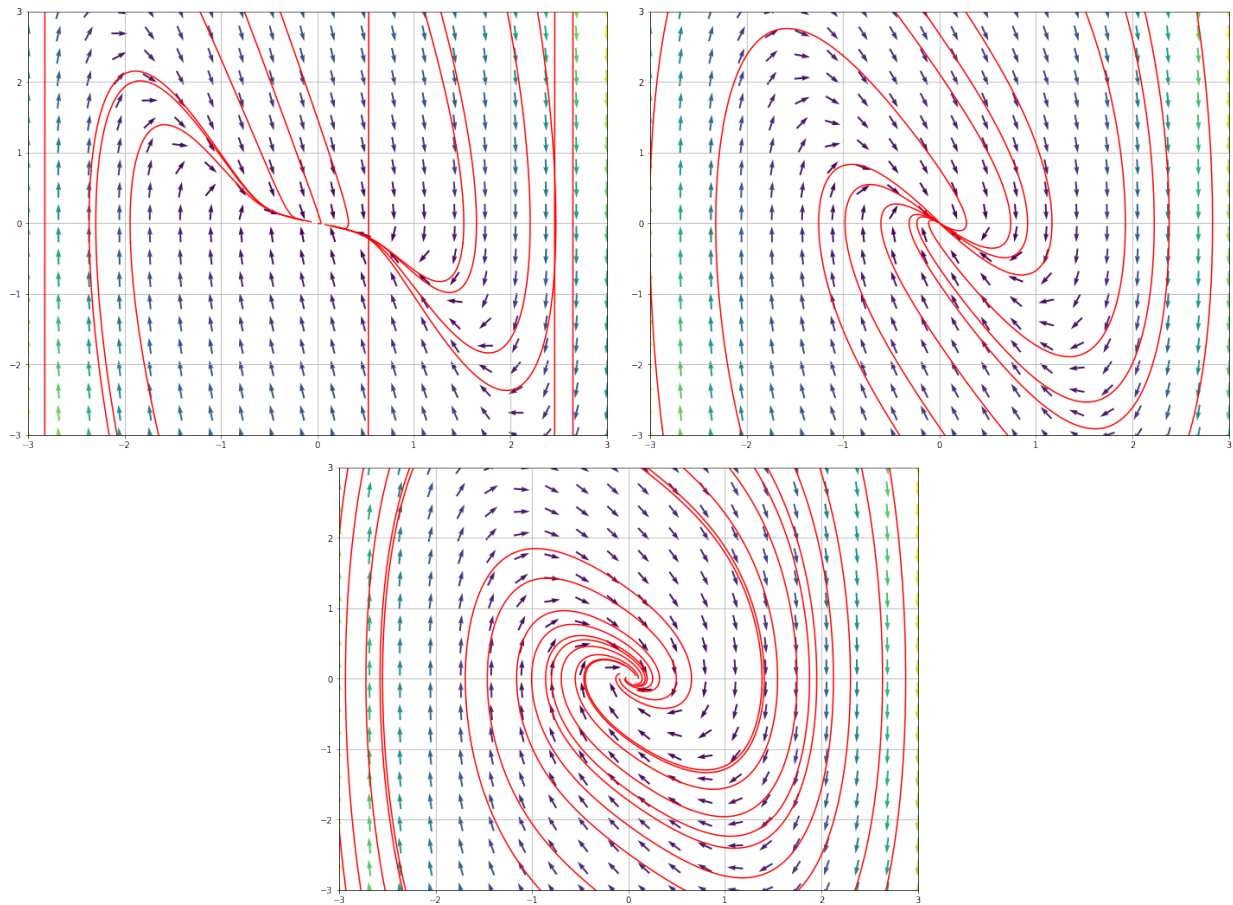


Figure 5: Phase plane for (a)  $\mu > 1$  (top left),  $\mu = 1$  (top right),  $0 < \mu < 1$  (bottom)

(b) Let  $x_1 = u$  and  $x_2 = \dot{u}$ , then system can be expressed in the form of

$$\begin{cases} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -2\mu x_2 - x_1 + x_1^3 \end{cases}$$

The elements  $a, b, c, d$  are

$$\begin{aligned} a &= \frac{\partial \dot{x}_1}{\partial x_1} = 0 \\ b &= \frac{\partial \dot{x}_1}{\partial x_2} = 1 \\ c &= \frac{\partial \dot{x}_2}{\partial x_1} = 3x_1^2 - 1 \\ d &= \frac{\partial \dot{x}_2}{\partial x_2} = -2\mu \end{aligned}$$

which gives

$$A = \begin{bmatrix} 0 & 1 \\ 3x_1^2 - 1 & -2\mu \end{bmatrix}.$$

The equilibrium points for this system is  $(x_{1e}, x_{2e}) = (-1, 0), (0, 0), (1, 0)$ . At these points the system matrix  $A$  becomes

$$A|_{(-1,0)} = \begin{bmatrix} 0 & 1 \\ 2 & -2\mu \end{bmatrix}$$

$$A|_{(0,0)} = \begin{bmatrix} 0 & 1 \\ -1 & -2\mu \end{bmatrix}$$

$$A|_{(1,0)} = \begin{bmatrix} 0 & 1 \\ 2 & -2\mu \end{bmatrix}.$$

For equilibrium point  $(-1, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu - \sqrt{\mu^2 + 2}, \quad \lambda_2 = -\mu + \sqrt{\mu^2 + 2}$$

$$v_1 = \begin{bmatrix} \frac{1}{2}\mu - \frac{1}{2}\sqrt{\mu^2 + 2} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} \frac{1}{2}\mu + \frac{1}{2}\sqrt{\mu^2 + 2} \\ 1 \end{bmatrix}.$$

For equilibrium point  $(0, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu + \sqrt{\mu^2 - 1}, \quad \lambda_2 = -\mu - \sqrt{\mu^2 - 1}$$

$$v_1 = \begin{bmatrix} -\mu - \sqrt{\mu^2 - 1} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} -\mu + \sqrt{\mu^2 - 1} \\ 1 \end{bmatrix}$$

For equilibrium point  $(1, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu - \sqrt{\mu^2 + 2}, \quad \lambda_2 = -\mu + \sqrt{\mu^2 + 2}$$

$$v_1 = \begin{bmatrix} \frac{1}{2}\mu - \frac{1}{2}\sqrt{\mu^2 + 2} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} \frac{1}{2}\mu + \frac{1}{2}\sqrt{\mu^2 + 2} \\ 1 \end{bmatrix}.$$

Thus, the equilibrium points are

for  $(0, 0)$

→ if  $\mu > 1$ : a stable node

→ if  $\mu = 1$ : a stable singular node

→ if  $0 < \mu < 1$ : a stable spiral/focus

for  $(-1, 0)$  and  $(1, 0)$

→ it is an unstable saddle point

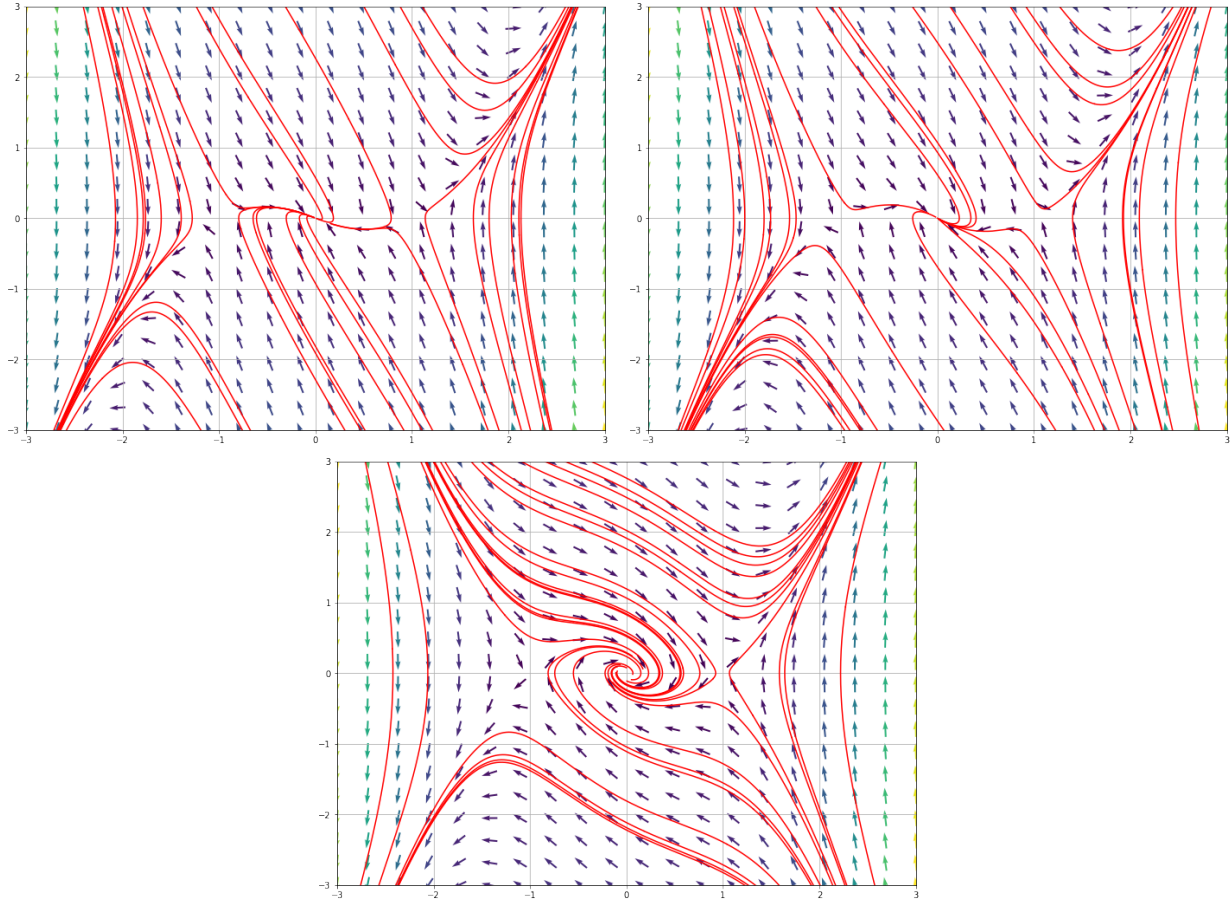


Figure 6: Phase plane for (b)  $\mu > 1$  (top left),  $\mu = 1$  (top right),  $0 < \mu < 1$  (bottom)

(c) Let  $x_1 = u$  and  $x_2 = \dot{u}$ , then system can be expressed in the form of

$$\begin{cases} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -2\mu x_2 + x_1 - x_1^3 \end{cases}$$

The elements  $a, b, c, d$  are

$$\begin{aligned} a &= \frac{\partial \dot{x}_1}{\partial x_1} = 0 \\ b &= \frac{\partial \dot{x}_1}{\partial x_2} = 1 \\ c &= \frac{\partial \dot{x}_2}{\partial x_1} = -3x_1^2 + 1 \\ d &= \frac{\partial \dot{x}_2}{\partial x_2} = -2\mu \end{aligned}$$

which gives

$$A = \begin{bmatrix} 0 & 1 \\ -3x_1^2 + 1 & -2\mu \end{bmatrix}.$$

The equilibrium points for this system is  $(x_{1e}, x_{2e}) = (-1, 0), (0, 0), (1, 0)$ . At these points the system matrix  $A$  becomes

$$A|_{(-1,0)} = \begin{bmatrix} 0 & 1 \\ -2 & -2\mu \end{bmatrix}$$

$$A|_{(0,0)} = \begin{bmatrix} 0 & 1 \\ 1 & -2\mu \end{bmatrix}$$

$$A|_{(1,0)} = \begin{bmatrix} 0 & 1 \\ -2 & -2\mu \end{bmatrix}.$$

For equilibrium point  $(-1, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu - \sqrt{\mu^2 - 2}, \quad \lambda_2 = -\mu + \sqrt{\mu^2 - 2}$$

$$v_1 = \begin{bmatrix} -\frac{1}{2}\mu + \frac{1}{2}\sqrt{\mu^2 - 2} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} -\frac{1}{2}\mu - \frac{1}{2}\sqrt{\mu^2 - 2} \\ 1 \end{bmatrix}.$$

For equilibrium point  $(0, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu + \sqrt{\mu^2 + 1}, \quad \lambda_2 = -\mu - \sqrt{\mu^2 + 1}$$

$$v_1 = \begin{bmatrix} \mu - \sqrt{\mu^2 + 1} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} \mu + \sqrt{\mu^2 + 1} \\ 1 \end{bmatrix}$$

For equilibrium point  $(1, 0)$  the eigenvalues and eigenvectors become

$$\lambda_1 = -\mu - \sqrt{\mu^2 - 2}, \quad \lambda_2 = -\mu + \sqrt{\mu^2 - 2}$$

$$v_1 = \begin{bmatrix} -\frac{1}{2}\mu + \frac{1}{2}\sqrt{\mu^2 - 2} \\ 1 \end{bmatrix}, \quad v_2 = \begin{bmatrix} -\frac{1}{2}\mu - \frac{1}{2}\sqrt{\mu^2 - 2} \\ 1 \end{bmatrix}.$$

Thus, the equilibrium points are

for  $(-1, 0)$  and  $(1, 0)$

→ if  $\mu > \sqrt{2}$ : a stable node

→ if  $\mu = \sqrt{2}$ : a stable singular node

→ if  $0 < \mu < \sqrt{2}$ : a stable spiral/focus

for  $(0, 0)$

→ it is an unstable saddle point

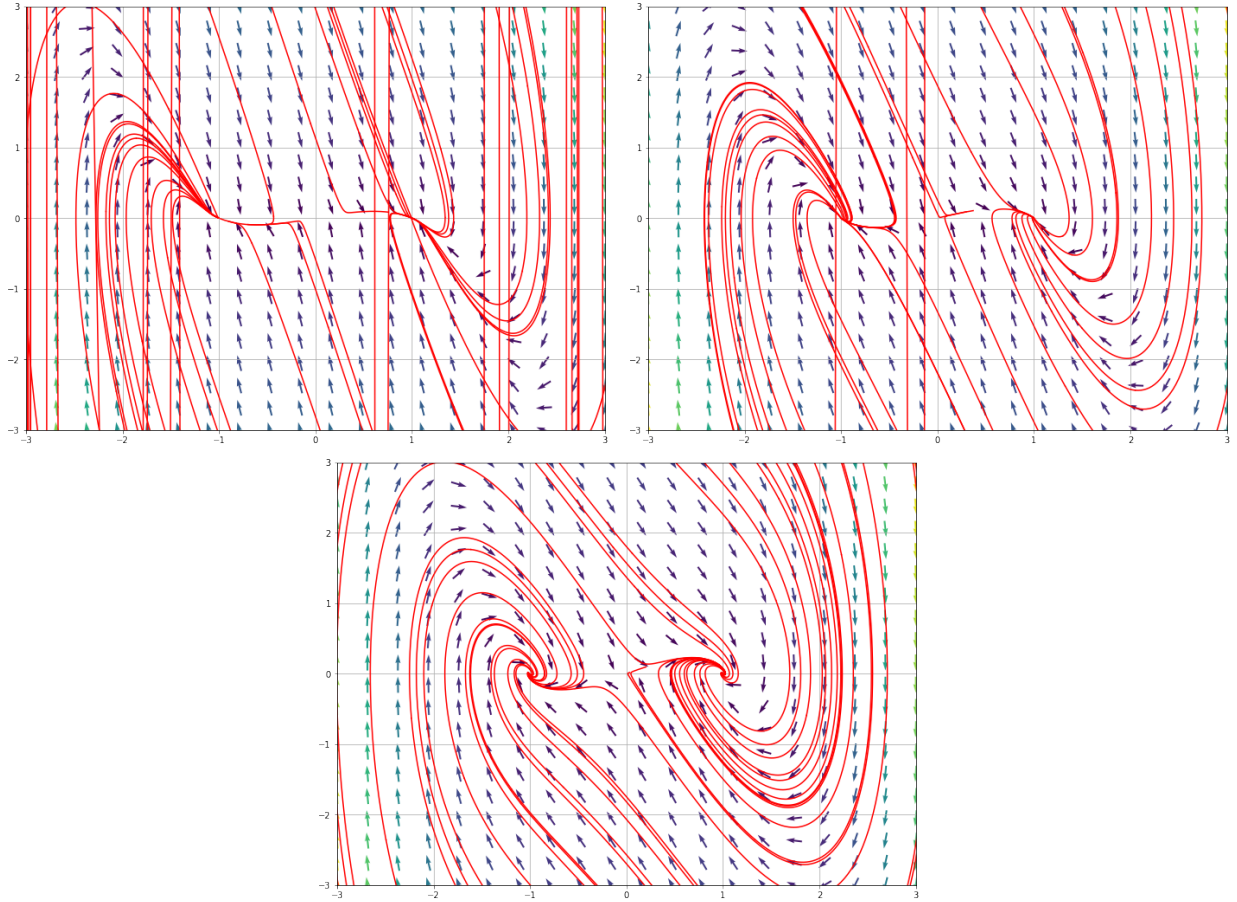


Figure 7: Phase plane for (c)  $\mu > \sqrt{2}$  (top left),  $\mu = \sqrt{2}$  (top right),  $0 < \mu < \sqrt{2}$  (bottom)



(d) Let  $x_1 = u$  and  $x_2 = \dot{u}$ , then system can be expressed in the form of

$$\begin{cases} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= -2\mu x_2 + x_1 + x_1^3 \end{cases}$$

The elements  $a, b, c, d$  are

$$\begin{aligned} a &= \frac{\partial \dot{x}_1}{\partial x_1} = 0 \\ b &= \frac{\partial \dot{x}_1}{\partial x_2} = 1 \\ c &= \frac{\partial \dot{x}_2}{\partial x_1} = 3x_1^2 + 1 \\ d &= \frac{\partial \dot{x}_2}{\partial x_2} = -2\mu \end{aligned}$$

which gives

$$A = \begin{bmatrix} 0 & 1 \\ 3x_1^2 + 1 & -2\mu \end{bmatrix}.$$

The equilibrium points for this system is  $x_{1e} = x_{2e} = 0$ . At this point the system matrix  $A$  becomes

$$A_e = \begin{bmatrix} 0 & 1 \\ 1 & -2\mu \end{bmatrix}.$$

and the eigenvalues and eigenvectors become

$$\begin{aligned} \lambda_1 &= -\mu - \sqrt{\mu^2 + 1}, & \lambda_2 &= -\mu + \sqrt{\mu^2 + 1} \\ v_1 &= \begin{bmatrix} \mu - \sqrt{\mu^2 + 1} \\ 1 \end{bmatrix}, & v_2 &= \begin{bmatrix} \mu + \sqrt{\mu^2 + 1} \\ 1 \end{bmatrix} \end{aligned}$$

Thus, the equilibrium points are

for  $(0, 0)$   
→ it is an unstable saddle point

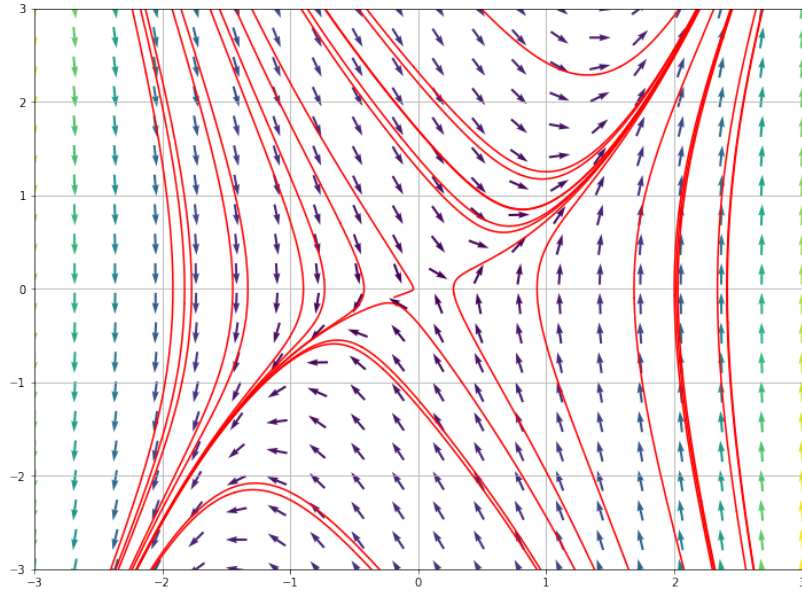


Figure 8: Phase plane for (d)  $\mu = 0.8$  ( $\mu$  could be any positive value)

## Appendix

### 3.1 Problem 1: MATLAB Code

```
1 % AE6444 HW4 Problem 1 MATLAB
2 % Tomoki Koike
3 close all; clear all; clc;
4 %%
5 % Define the system
6 syms x_1 x_2 Omega k m n l theta theta_dot
7 assume(n, {'integer', 'positive'});
8
9 T = 1/8*m*l^2*theta_dot^2*cos(theta)^2 + 1/8*m*l^2*Omega^2*sin(theta)^2;
10 V = 1/4*k*l^2 * (1 - cos(theta))^2;
11 L = T - V;
12
13 x1dot = x_2
14 x2dot = x_2^2 * tan(x_1) + Omega^2 * tan(x_1) - 2*k/m*(1-cos(x_1))*sec(x_1)*
    tan(x_1);
15 %%
16 % System matrix: A
17 a = diff(x1dot, x_1);
18 b = diff(x1dot, x_2);
19 c = diff(x2dot, x_1)
20 d = diff(x2dot, x_2)
21 A = [a, b; c, d];
22 %%
23 % Equilibrium points
24 x1e_1 = n*pi;
25 x1e_2 = acos(2*k / (2*k + m*Omega^2));
26 x2e = 0;
27 %%
28 % First possible solution
29 A1 = subs(A, [x_1, x_2], [x1e_1, x2e]);
30 A1_odd = subs(A1, n, 1) % odd n value
31 A1_even = subs(A1, n, 2) % even n value
32 %%
33 % Eigenvalues for first possible solution
34 [v_even, d_even] = eig(A1_even)
35 [v_odd, d_odd] = eig(A1_odd)
36 %%
37 % Second possible solution
38 A2 = simplify(subs(A, [x_1, x_2], [x1e_2, x2e]))
39 %%
```

```

40 % Eigenvalues for second possible solution
41 [v2, d2] = eig(A2)
42 %%
43 % Hamiltonian
44
45 H = L - diff(L, theta_dot)*theta_dot;
46 %%
47 a_h = diff(diff(H, theta), theta_dot)
48 b_h = diff(diff(H, theta_dot), theta_dot)
49 c_h = -diff(diff(H, theta), theta)
50 d_h = -diff(diff(H, theta_dot), theta)
51 %%
52 p = a_h + d_h
53 q = a_h * d_h - b_h * c_h
54 %%
55 % (n\pi, 0)
56 subs(q, [theta, theta_dot], [1*pi, 0])
57 subs(q, [theta, theta_dot], [2*pi, 0])
58
59 % (<>, 0)
60 simplify(subs(q, [theta, theta_dot], [x1e_2, 0]))

```

### 3.2 Problem 2: MATLAB Code

```

1 % AE6444 HW4 Problem 2 MATLAB
2 % Tomoki Koike
3 close all; clear all; clc;
4 %%
5 syms mu u(t) x_1 x_2
6 assume(mu, 'positive');
7
8 % System
9 x1dot = x_2;
10 x2dot = -2*mu*x_2 - x_1 - x_1^3;
11
12 % System matrix: A
13 a = diff(x1dot, x_1);
14 b = diff(x1dot, x_2);
15 c = diff(x2dot, x_1)
16 d = diff(x2dot, x_2)
17 A = [a, b; c, d]
18
19 % Equilibrium points

```

```

20 x1e = 0;
21 x2e = 0;
22
23 % A matrix for equilibrium point
24 A1 = subs(A, [x_1 x_2], [x1e x2e])
25
26 % Eigenvalues and eigenvectors
27 [V, D] = eig(A1)
28 %%
29 % System
30 x1dot = x_2;
31 x2dot = -2*mu*x_2 - x_1 + x_1^3;
32
33 % System matrix: A
34 a = diff(x1dot, x_1);
35 b = diff(x1dot, x_2);
36 c = diff(x2dot, x_1)
37 d = diff(x2dot, x_2)
38 A = [a, b; c, d]
39
40 % Equilibrium points
41 x1e_1 = -1;
42 x2e_1 = 0;
43 x1e_2 = 0;
44 x2e_2 = 0;
45 x1e_3 = 1;
46 x2e_3 = 0;
47
48 % A matrix for equilibrium point
49 A1 = subs(A, [x_1 x_2], [x1e_1 x2e_1])
50 A2 = subs(A, [x_1 x_2], [x1e_2 x2e_2])
51 A3 = subs(A, [x_1 x_2], [x1e_3 x2e_3])
52
53 % Eigenvalues and eigenvectors
54 [V1, D1] = eig(A1)
55 [V2, D2] = eig(A2)
56 [V3, D3] = eig(A3)
57 %%
58 % System
59 x1dot = x_2;
60 x2dot = -2*mu*x_2 + x_1 - x_1^3;
61
62 % System matrix: A
63 a = diff(x1dot, x_1);
64 b = diff(x1dot, x_2);

```

```

65 c = diff(x2dot, x_1)
66 d = diff(x2dot, x_2)
67 A = [a, b; c, d]
68
69 % Equilibrium points
70 x1e_1 = -1;
71 x2e_1 = 0;
72 x1e_2 = 0;
73 x2e_2 = 0;
74 x1e_3 = 1;
75 x2e_3 = 0;
76
77 % A matrix for equilibrium point
78 A1 = subs(A, [x_1 x_2], [x1e_1 x2e_1])
79 A2 = subs(A, [x_1 x_2], [x1e_2 x2e_2])
80 A3 = subs(A, [x_1 x_2], [x1e_3 x2e_3])
81
82 % Eigenvalues and eigenvectors
83 [V1, D1] = eig(A1)
84 [V2, D2] = eig(A2)
85 [V3, D3] = eig(A3)
86 %%
87 % System
88 x1dot = x_2;
89 x2dot = -2*mu*x_2 + x_1 + x_1^3;
90
91 % System matrix: A
92 a = diff(x1dot, x_1);
93 b = diff(x1dot, x_2);
94 c = diff(x2dot, x_1)
95 d = diff(x2dot, x_2)
96 A = [a, b; c, d]
97
98 % Equilibrium points
99 x1e = 0;
100 x2e = 0;
101
102 % A matrix for equilibrium point
103 A1 = subs(A, [x_1 x_2], [x1e x2e])
104
105 % Eigenvalues and eigenvectors
106 [V, D] = eig(A1)

```

### 3.3 Problem 2: Python Code

---

```
1  import matplotlib.pyplot as plt
2  import numpy as np
3  from scipy.integrate import solve_ivp, DOP853
4  from typing import List
5
6  # System ODE
7  def duff1(t, x, mu):
8      return [x[1], -2*mu*x[1] - x[0] - x[0]**3]
9
10 def duff2(t, x, mu):
11     return [x[1], -2*mu*x[1] - x[0] + x[0]**3]
12
13 def duff3(t, x, mu):
14     return [x[1], -2*mu*x[1] + x[0] - x[0]**3]
15
16 def duff4(t, x, mu):
17     return [x[1], -2*mu*x[1] + x[0] + x[0]**3]
18
19 def solve_diffreq(func, t, tspan, ic, parameters={}, algorithm='DOP853',
    ↪ stepsize=np.inf):
20     return solve_ivp(fun=func, t_span=tspan, t_eval=t, y0=ic, method=algorithm,
21                       args=tuple(parameters.values()), atol=1e-8, rtol=1e-5,
22                       ↪ max_step=stepsize)
23
24 def phasePlane(x1, x2, func, params):
25     X1, X2 = np.meshgrid(x1, x2) # create grid
26     u, v = np.zeros(X1.shape), np.zeros(X2.shape)
27     NI, NJ = X1.shape
28     for i in range(NI):
29         for j in range(NJ):
30             x = X1[i, j]
31             y = X2[i, j]
32             dx = func(0, (x, y), *params.values()) # compute values on grid
33             u[i, j] = dx[0]
34             v[i, j] = dx[1]
35
36     M = np.hypot(u, v)
37     u /= M
38     v /= M
39     return X1, X2, u, v, M
40
41 def DEplot(sys: object, tspan: tuple, x0: List[List[float]],
```

```

40         x: np.ndarray, y: np.ndarray, params: dict):
41     if len(tspan) != 3:
42         raise Exception('tspan should be tuple of size 3: (min, max, number of
43             ↪ points).')
44     # Set up the figure the way we want it to look
45     plt.figure(figsize=(12, 9))
46
47     X1, X2, dx1, dx2, M = phasePlane(
48         x, y, sys, params
49     )
50
51     # Quiver plot
52     plt.quiver(X1, X2, dx1, dx2, M, scale=None, pivot='mid')
53     plt.grid()
54
55     t1 = np.linspace(0, tspan[0], tspan[2])
56     t2 = np.linspace(0, tspan[1], tspan[2])
57     if min(tspan) < 0:
58         t_span1 = (np.max(t1), np.min(t1))
59     else:
60         t_span1 = (np.min(t1), np.max(t1))
61     t_span2 = (np.min(t2), np.max(t2))
62     for x0i in x0:
63         sol1 = solve_diffeq(sys, t1, t_span1, x0i, params)
64         plt.plot(sol1.y[0, :], sol1.y[1, :], '-r')
65         sol2 = solve_diffeq(sys, t2, t_span2, x0i, params)
66         plt.plot(sol2.y[0, :], sol2.y[1, :], '-r')
67
68     plt.xlim([np.min(x), np.max(x)])
69     plt.ylim([np.min(y), np.max(y)])
70     plt.show()
71
72     #  $\mu > 1$ 
73     # A stable node
74
75     x0 = np.random.uniform(-2.5, 2.5, (10, 2))
76
77     p = {'mu': 2}
78
79     x1 = np.linspace(-3, 3, 20)
80     x2 = np.linspace(-3, 3, 20)
81
82     DEplot(duff1, (-6, 6, 1000), x0, x1, x2, p)

```



```

83  #  $\mu = 1$ 
84  # A stable singular node
85
86  x0 = np.random.uniform(-2.5, 2.5, (10, 2))
87
88  p = {'mu': 1}
89
90  x1 = np.linspace(-3, 3, 20)
91  x2 = np.linspace(-3, 3, 20)
92
93  DEplot(duff1, (-6, 6, 1000), x0, x1, x2, p)
94
95  #  $0 < \mu < 1$ 
96  # A stable spiral/focus
97
98  x0 = np.random.uniform(-2.5, 2.5, (10, 2))
99
100 p = {'mu': 0.5}
101
102 x1 = np.linspace(-3, 3, 20)
103 x2 = np.linspace(-3, 3, 20)
104
105 DEplot(duff1, (-6, 6, 1000), x0, x1, x2, p)
106
107 #  $\mu > 1$ 
108 #  $(0, 0) \rightarrow$  a stable node
109 #  $(-1, 0), (1, 0) \rightarrow$  an unstable saddle point
110
111 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
112
113 p = {'mu': 1.2}
114
115 x1 = np.linspace(-3, 3, 20)
116 x2 = np.linspace(-3, 3, 20)
117
118 DEplot(duff2, (-6, 6, 1000), x0, x1, x2, p)
119
120 #  $\mu = 1$ 
121 #  $(0, 0) \rightarrow$  a stable singular node
122 #  $(-1, 0), (1, 0) \rightarrow$  an unstable saddle point
123
124 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
125
126 p = {'mu': 1}

```

```

127
128 x1 = np.linspace(-3, 3, 20)
129 x2 = np.linspace(-3, 3, 20)
130
131 DEplot(duff2, (-6, 6, 1000), x0, x1, x2, p)
132
133 # 0 < mu < 1
134 # (0, 0) -> a stable spiral/focus
135 # (-1, 0), (1, 0) -> an unstable saddle point
136
137 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
138
139 p = {'mu': 0.35}
140
141 x1 = np.linspace(-3, 3, 20)
142 x2 = np.linspace(-3, 3, 20)
143
144 DEplot(duff2, (-6, 6, 1000), x0, x1, x2, p)
145
146 # mu > sqrt(2)
147 # (0, 0) -> an unstable saddle point
148 # (-1, 0), (1, 0) -> a stable node
149
150 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
151
152 p = {'mu': 2}
153
154 x1 = np.linspace(-3, 3, 20)
155 x2 = np.linspace(-3, 3, 20)
156
157 DEplot(duff3, (-6, 6, 1000), x0, x1, x2, p)
158
159 # mu = sqrt(2)
160 # (0, 0) -> an unstable saddle point
161 # (-1, 0), (1, 0) -> a stable singular node
162
163 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
164
165 p = {'mu': np.sqrt(2)}
166
167 x1 = np.linspace(-3, 3, 20)
168 x2 = np.linspace(-3, 3, 20)
169
170 DEplot(duff3, (-6, 6, 1000), x0, x1, x2, p)

```

```

171
172 # 0 < mu < sqrt(2)
173 # (0, 0) -> an unstable saddle point
174 # (-1, 0), (1, 0) -> a stable sprial/focus
175
176 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
177
178 p = {'mu': 0.8}
179
180 x1 = np.linspace(-3, 3, 20)
181 x2 = np.linspace(-3, 3, 20)
182
183 DEplot(duff3, (-6, 6, 1000), x0, x1, x2, p)
184
185 # An unstable saddle point
186
187 x0 = np.random.uniform(-2.5, 2.5, (30, 2))
188
189 p = {'mu': 0.8}
190
191 x1 = np.linspace(-3, 3, 20)
192 x2 = np.linspace(-3, 3, 20)
193
194 DEplot(duff4, (-6, 6, 1000), x0, x1, x2, p)

```

---