

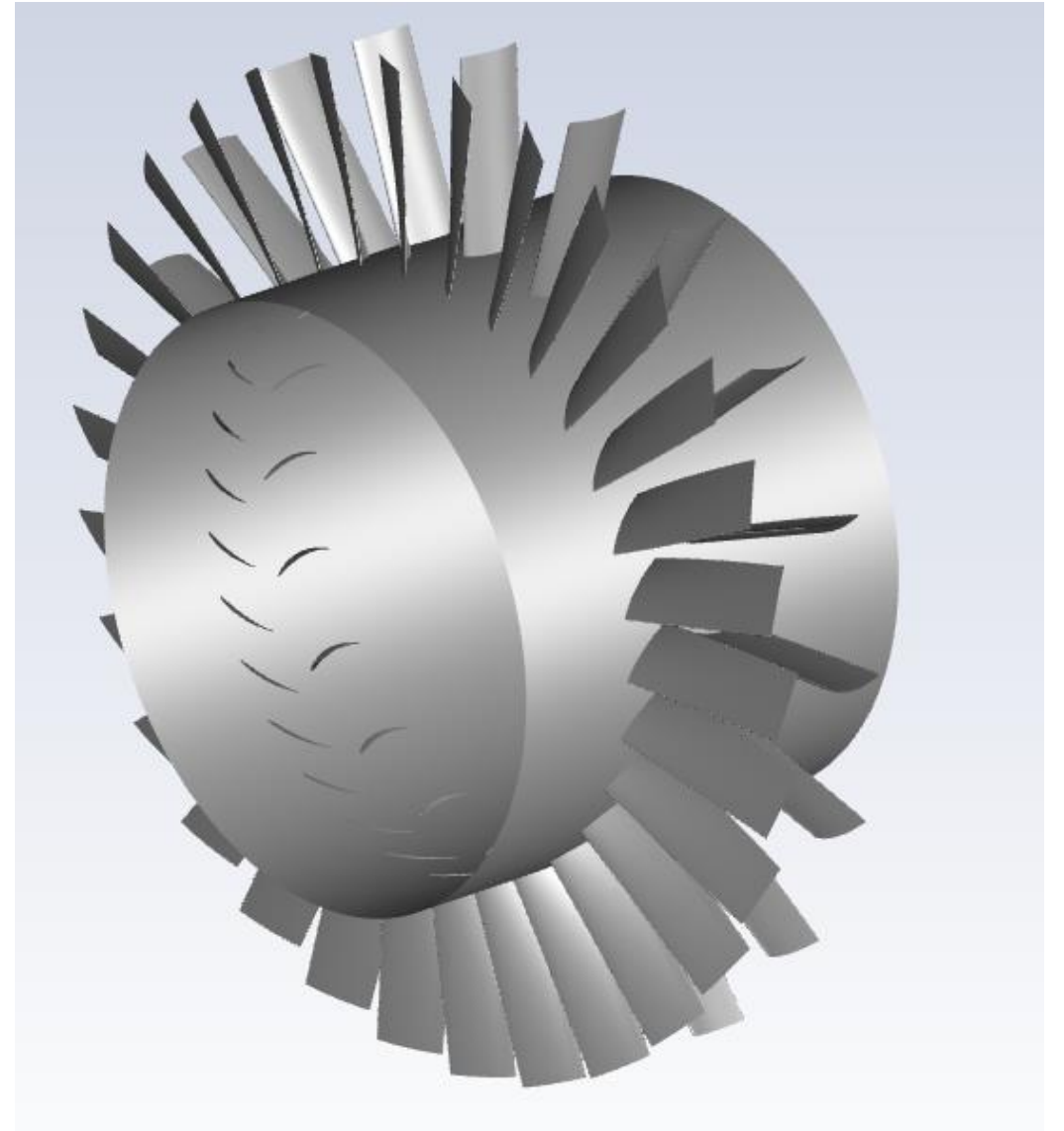
Workshop 04.2: Axial Fan Stage

Release 2020 R2



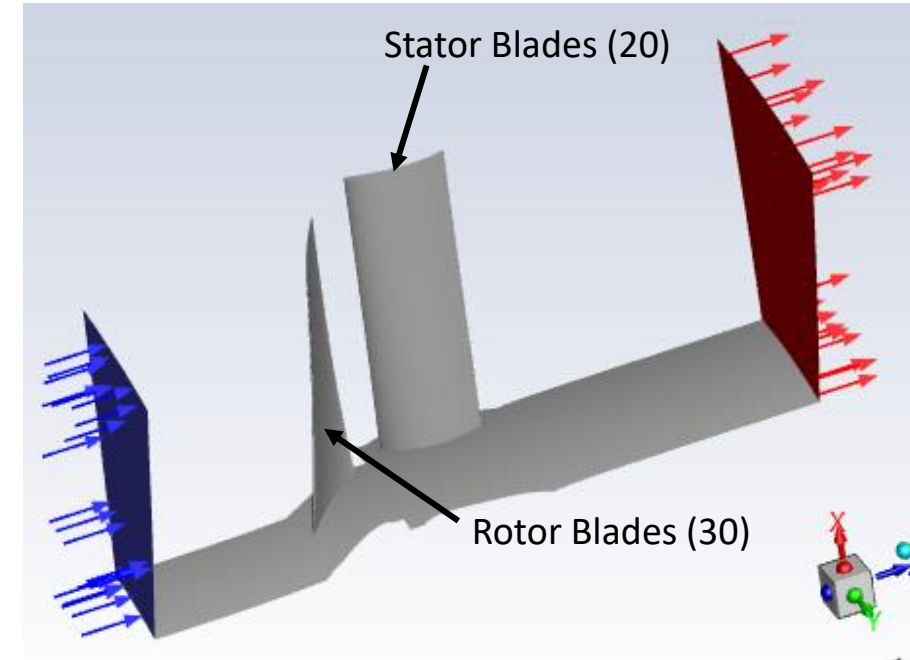
/ Introduction

- Workshop Description:
 - This Workshop deals with the Fluent setup and solution for an axial fan stage
- Learning Aims:
 - Setting up a steady stage calculation comprising a rotor and a stator
 - Defining a rotating frame
 - Applying rotational periodicity
 - Creating named expressions and report plots for monitoring the pressure rise and the power consumption of the fan rotor
 - Creating a Mixing Plane, General Turbo Interface
 - Solving and monitoring convergence
 - Visualizing the pressure distribution on the impeller walls and the relative velocity vectors at midspan



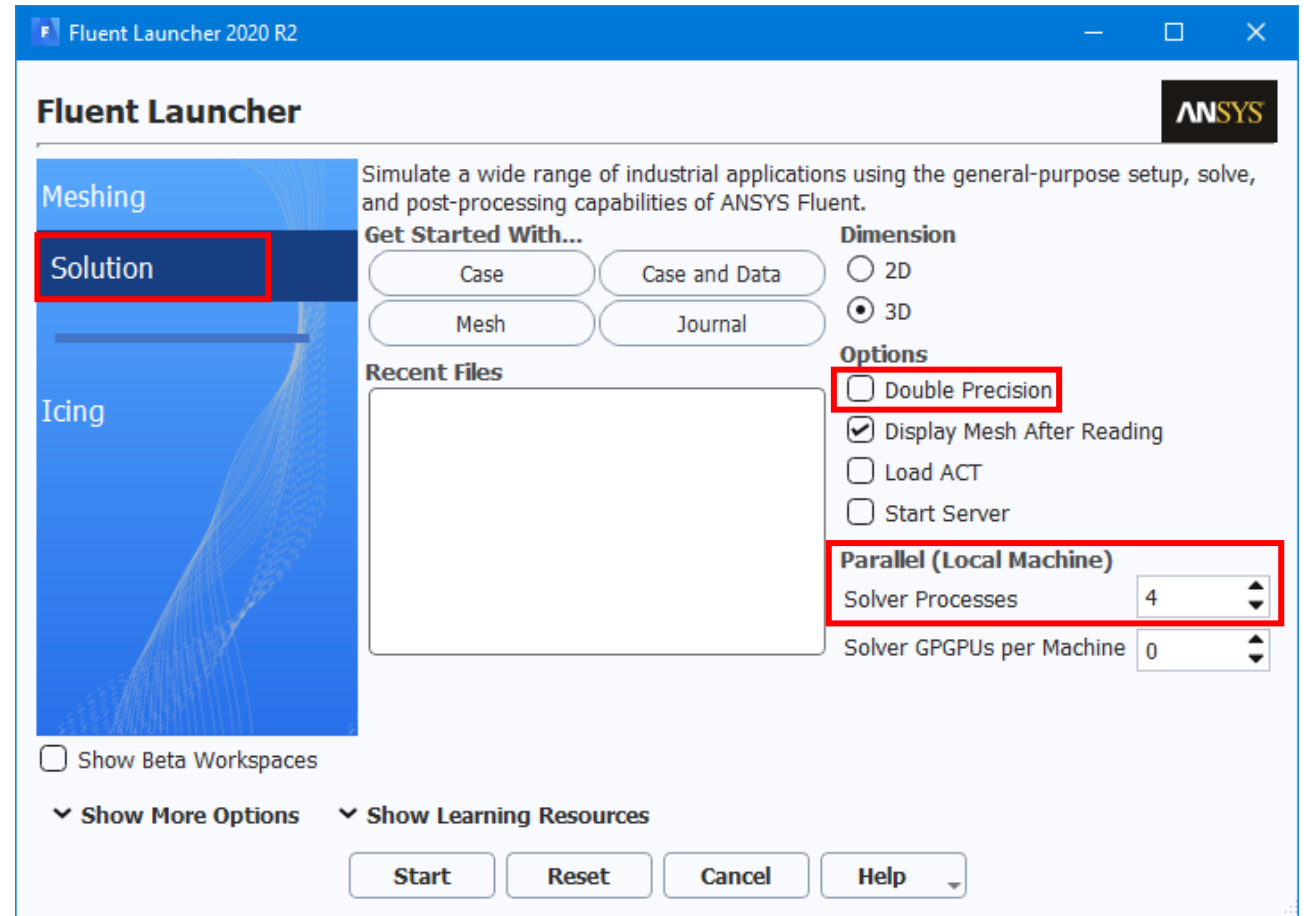
Fan Model

- A rotating component, followed by a stationary component
 - A moving reference frame is used to solve the rotating component
 - Due to rotational periodicity, we can reduce the problem size by modelling a single blade passage for the rotor and for the stator with periodic boundaries
- Fan data
 - Fluid = Air Ideal Gas
 - Operating Pressure = 1 bar = 100,000 Pa
 - Speed = 2880 rpm
 - Number of rotor blades = 30
 - Number of stator blades = 20
 - Axis of rotation = z-axis
 - Conditions
 - Inlet
 - $P_t = 1.0$ bar, $T_t = 288$ K
 - Outlet
 - Mass Flow Rate = 0.3 kg/s (one passage)



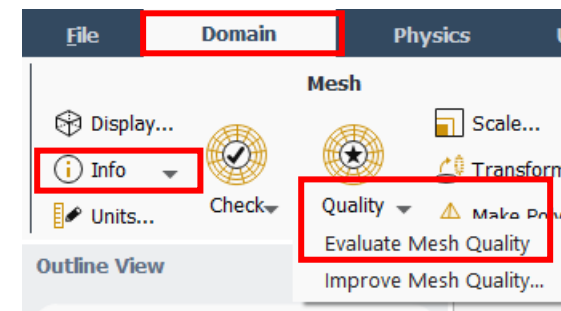
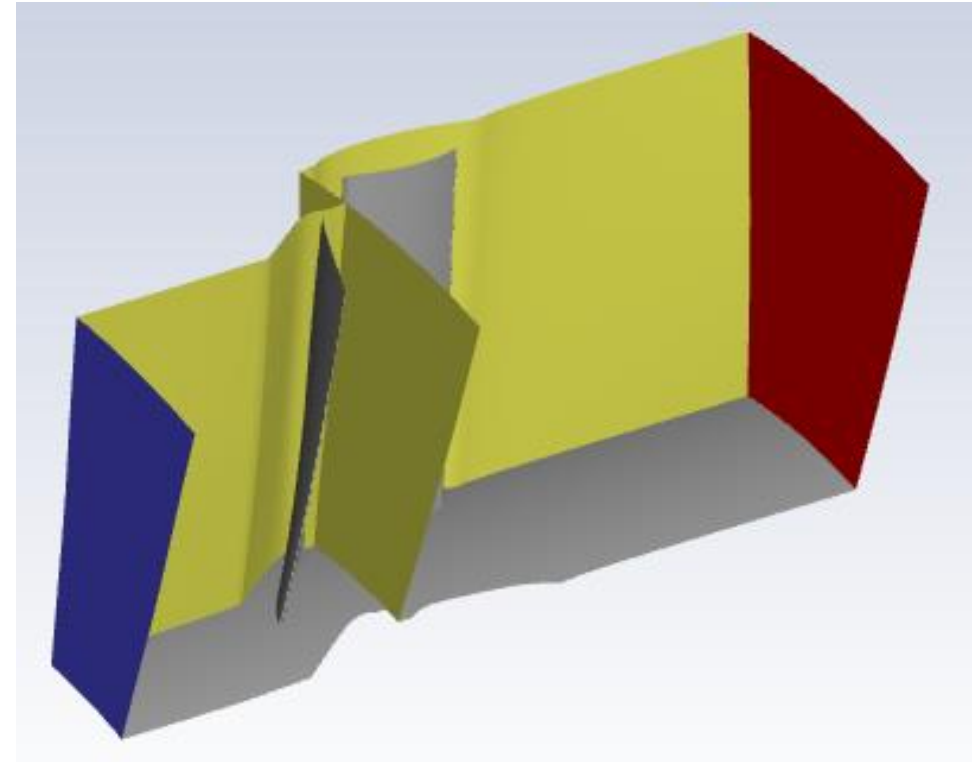
/ Start Fluent Launcher

- Start *Fluent Launcher* in *Solution* mode
- Do not check *Double Precision* as the Maximum Aspect Ratio is much smaller than 1000 (see next slide)
- Set the number of Processes for Parallel to 4
 - The mesh size for this case is approximately 97,000 cells (see next slide)
 - If you have enough Parallel licenses and more than 4 cores available, you may set up a number of up to 5 Processes (so that each Processor is solving for not much less than 20,000 cells)



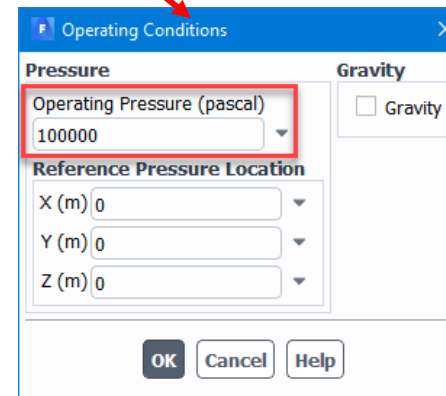
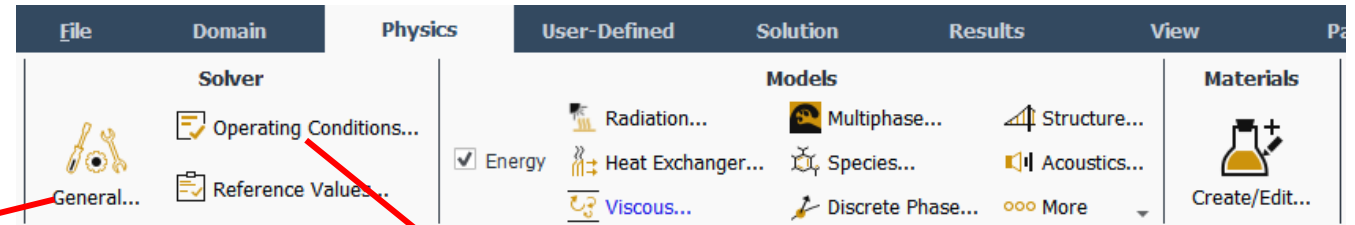
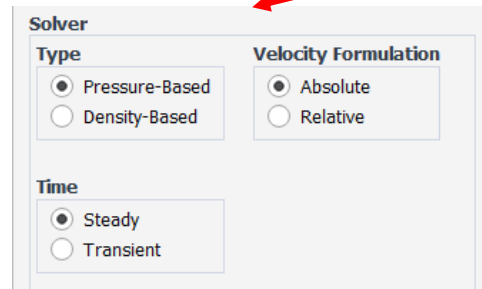
Fluent

- In the Fluent window read the mesh provided with the workshop inputs
 - File>Read>Mesh
 - Browse to file *AxialFanStage.msh*
- In the graphics window, you should see the single passage geometry of the axial fan stage as shown on the right
 - Note that in the image shown, the shroud boundary and one of the periodic boundaries were hidden (LMB to select a surface in the graphics window, RMB>Hide>Selected)
- The mesh comprises one rotor and one stator passage
- It is always a good practice to check the mesh size and the mesh quality in the *Mesh* group of the *Domain* tab
 - Info > Size will give you the number of cells in the Fluent Console, which is approximately 97,000
 - Quality > Evaluate Mesh Quality will show you a Maximum Aspect Ratio of $5.53e+02 < 1000$
 - This justifies the choice of starting Fluent in Single Precision



Physics: General & Operating Conditions

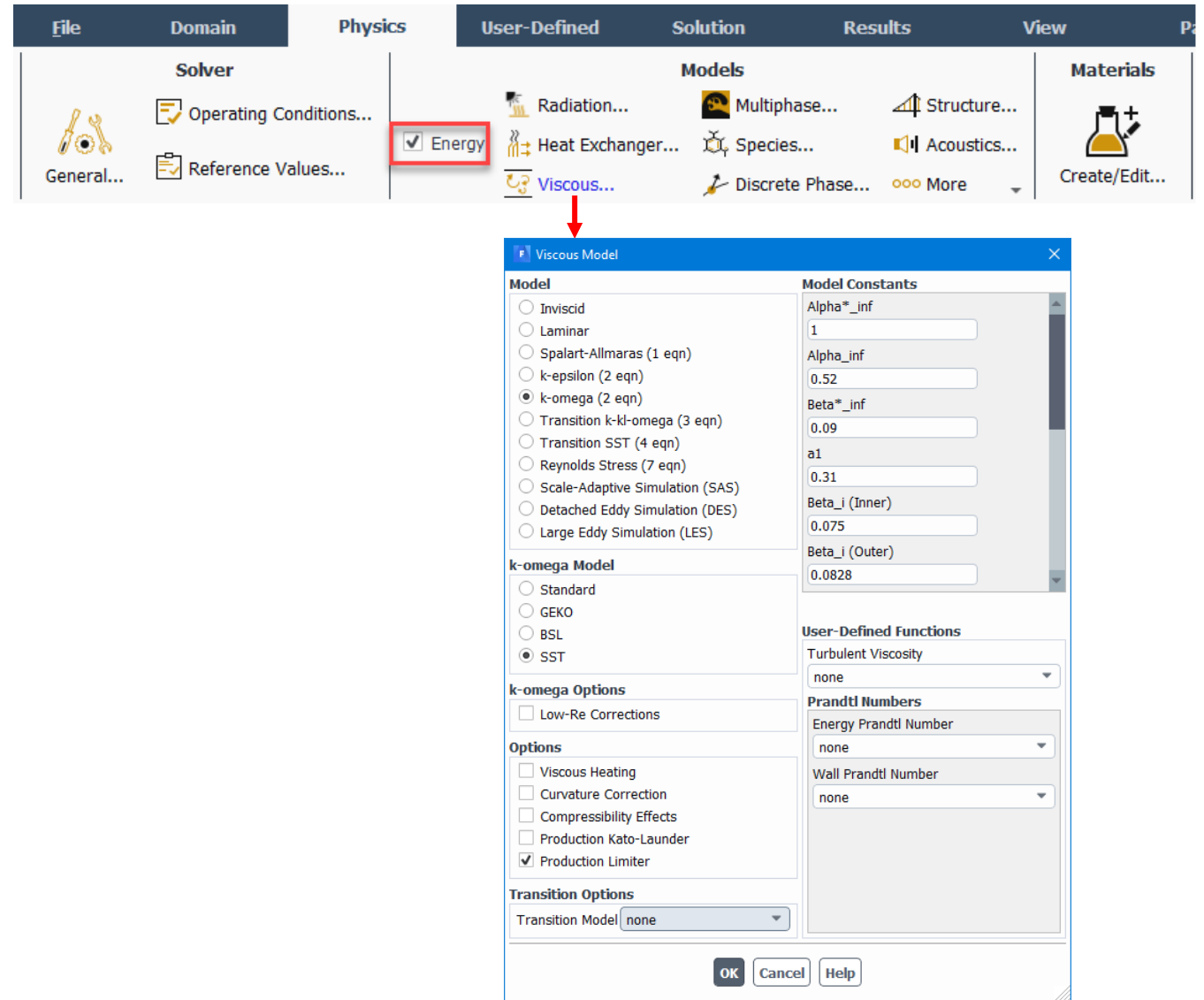
- Retain the default solver settings of *Pressure-Based* solver with *Absolute Velocity Formulation*



- In the *Operating Conditions* panel set the *Operating Pressure* to 100,000 (Pa)

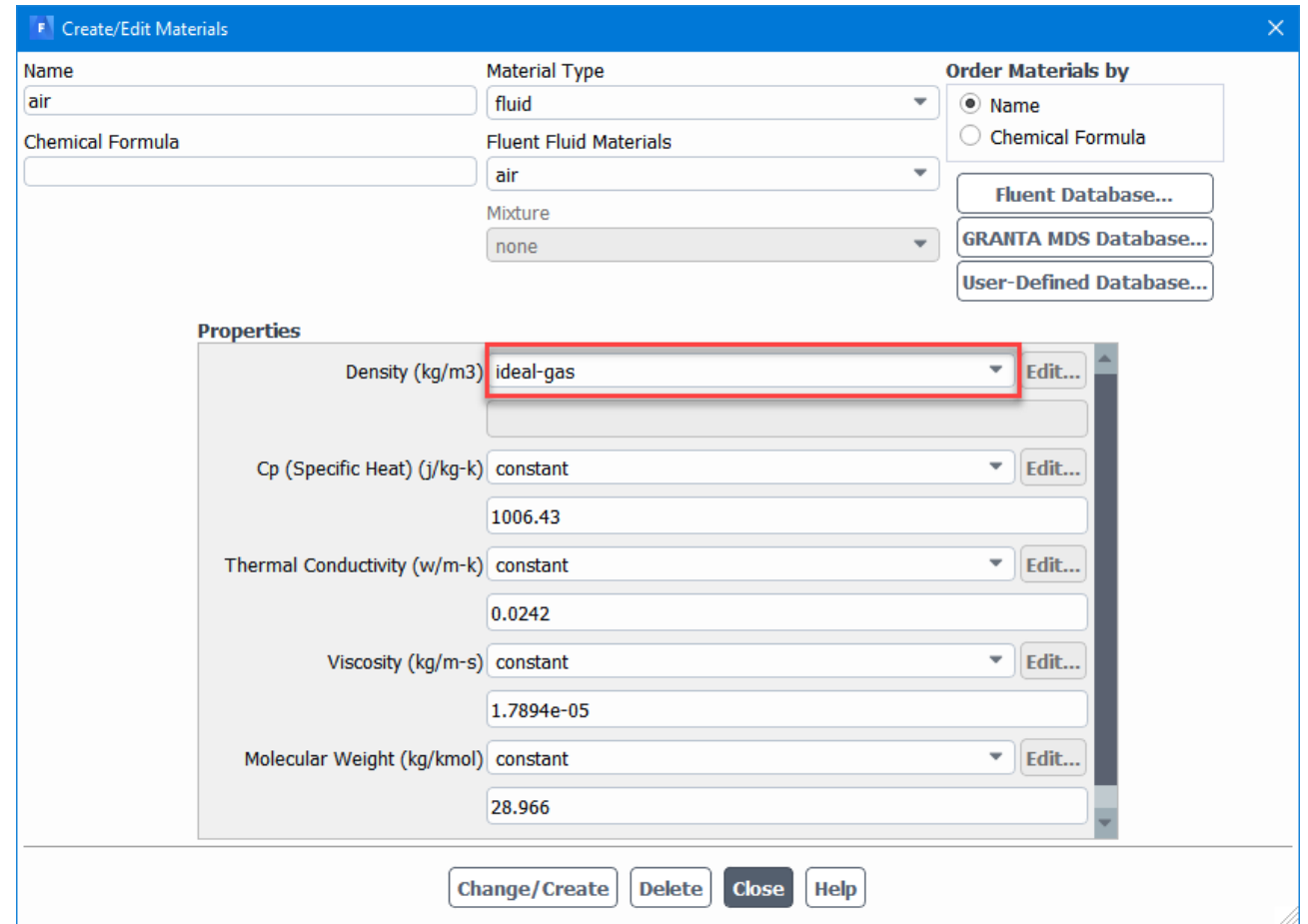
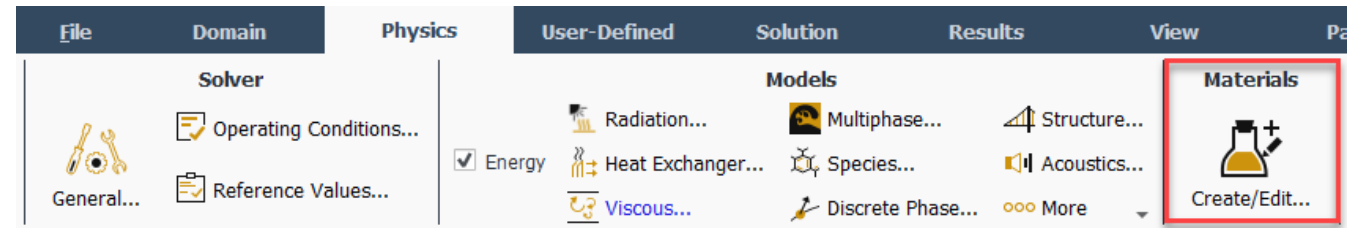
Physics: Energy and Viscous Model

- Enable the Energy equations
- Keep the default SST k-omega Viscous Model, which is the recommended turbulence model for turbomachinery simulations



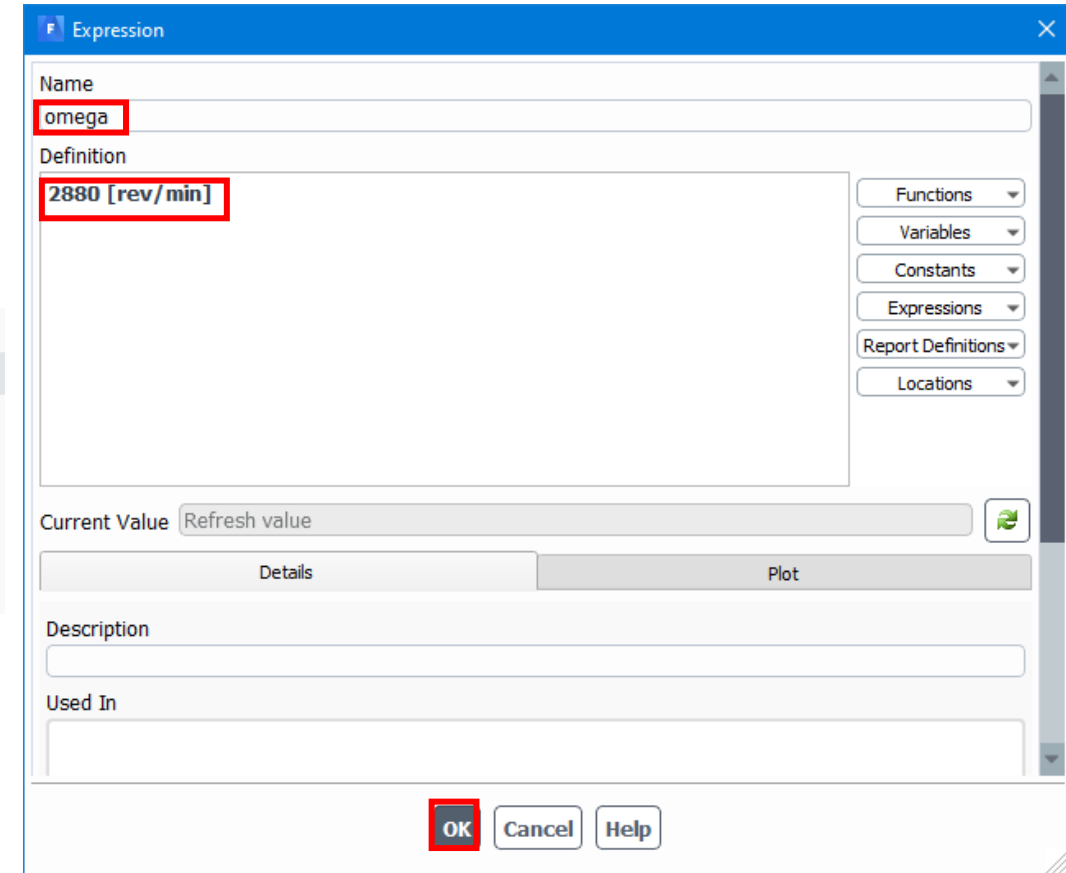
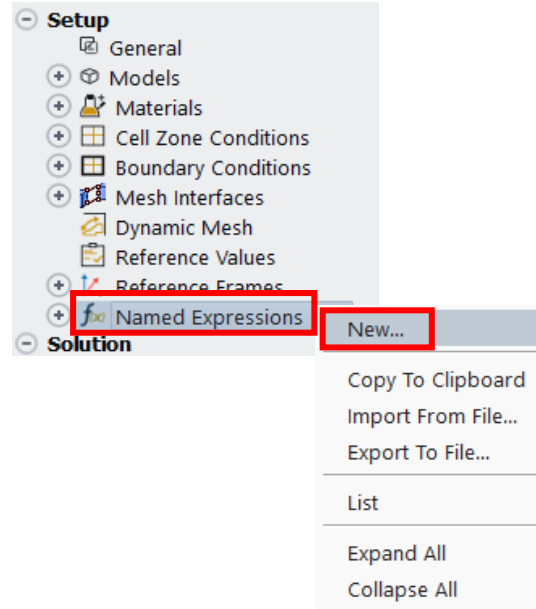
Physics: Materials

- The default material is Air with constant properties
 - We will need to change its Density to be a function of Temperature
- Click *Material > Create/Edit* in the *Physics* tab
 - Select *ideal-gas* from the *Density* drop-down list
 - Click *Change/Create* then *Close*



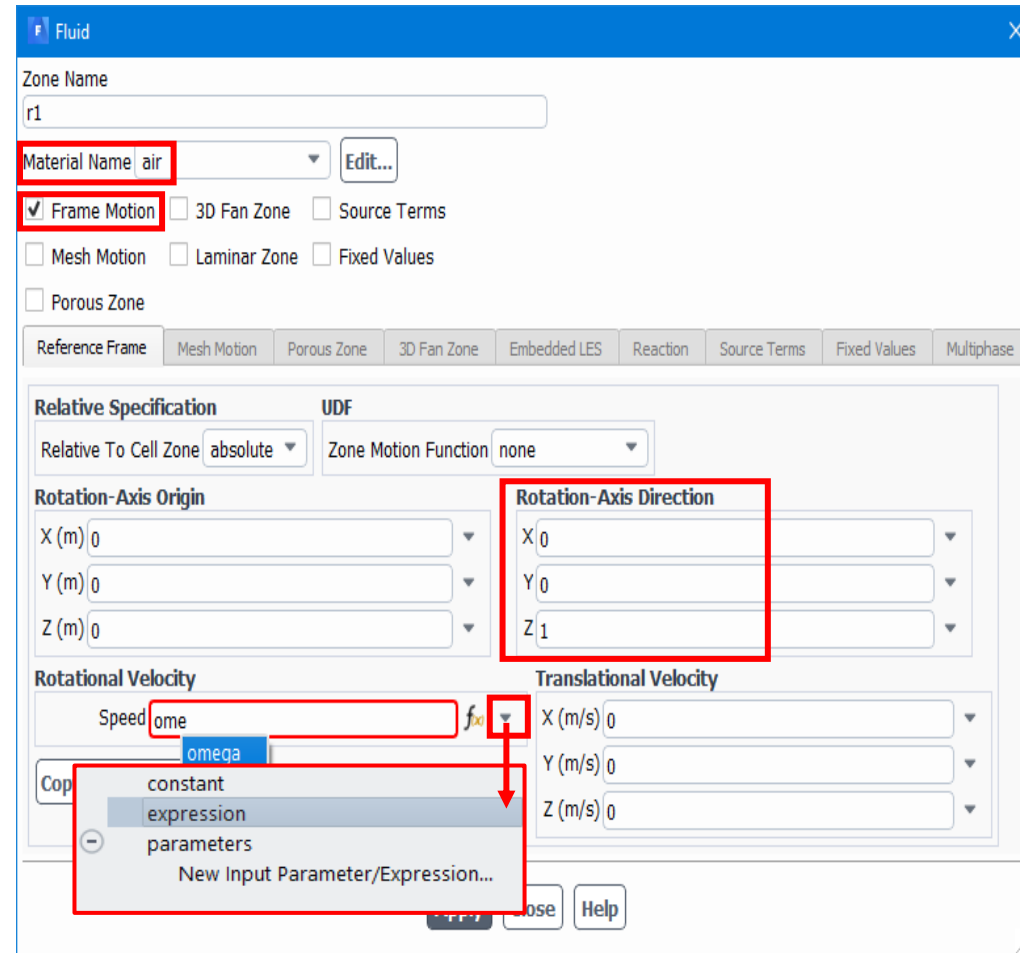
Create a Named Expression for Rotational Speed

- The fan rotor is rotating with a rotational speed of 2880 rpm
- We will need to set this in the conditions for the *r1* cell zone (see next slide)
- We are going to use a *Named Expression* for this
- In the *Outline*, *RMB* on *Named Expressions* and select *New...*
- In the *Expression* panel:
 - enter *omega* under *Name*,
 - *2880 [rev/min]* under *Definition*
 - Click *OK*



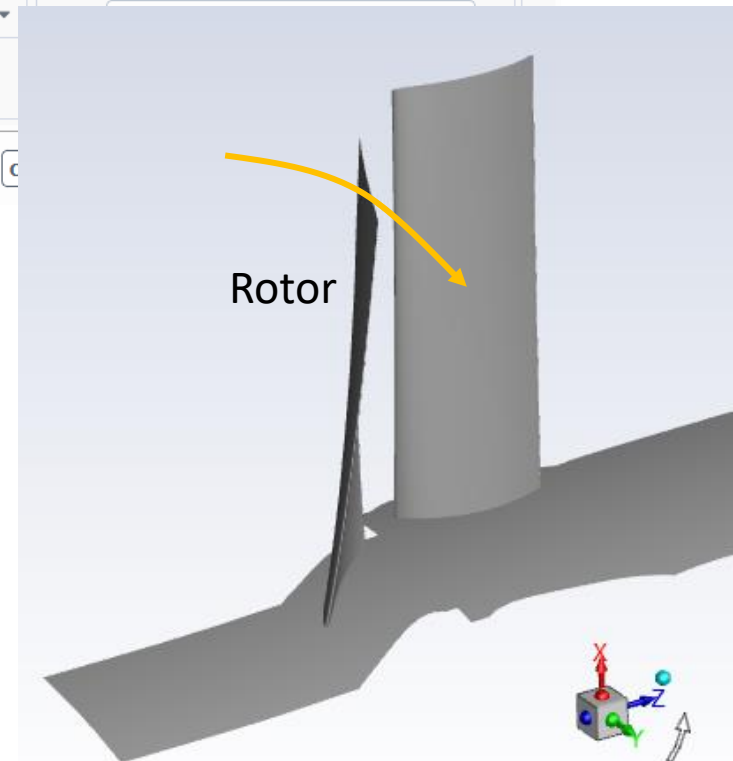
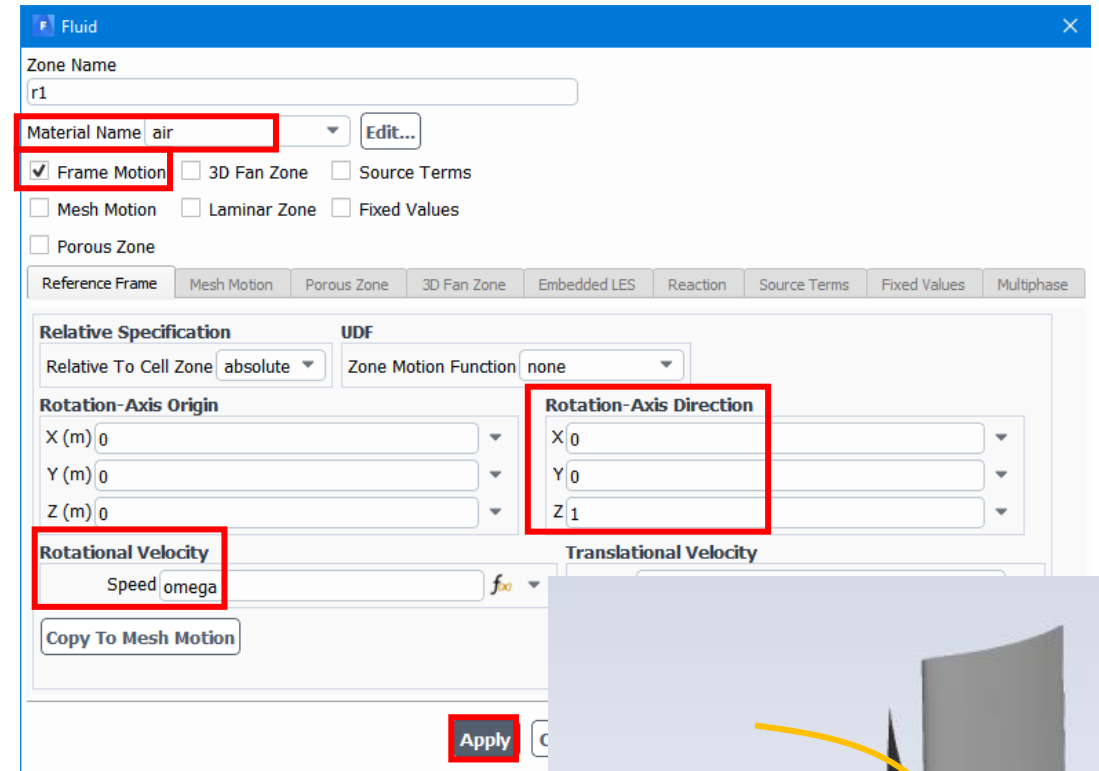
Physics: Cell Zone Conditions

- Edit the *r1* cell zone (rotor)
 - Leave the default *air* as *Material Name*
 - Enable *Frame Motion*
 - The default *Rotation-Axis Direction* is the z-axis and is suitable for this case
 - Using the drop-down list next to *Speed* under *Rotational Velocity*, set this to *expression*
 - In the box next to *Speed*, type *omega* (i.e., the name of the expression created in the previous slide)
 - After typing the first few letters of *omega*, you will see that the name *omega* is highlighted in a blue box. You may left-click on it to select it, or continue typing the complete expression name



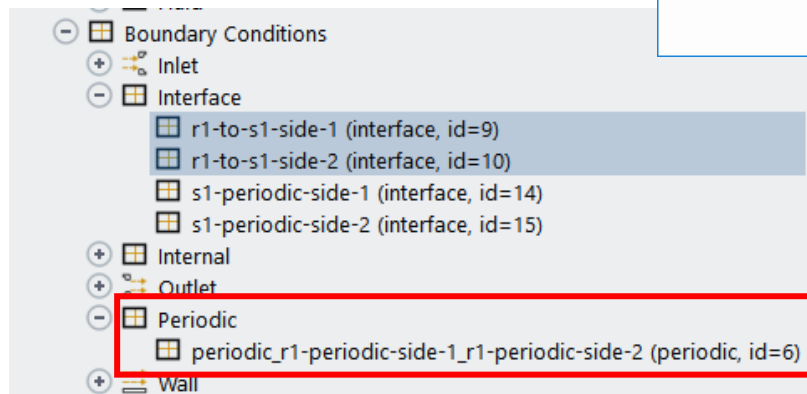
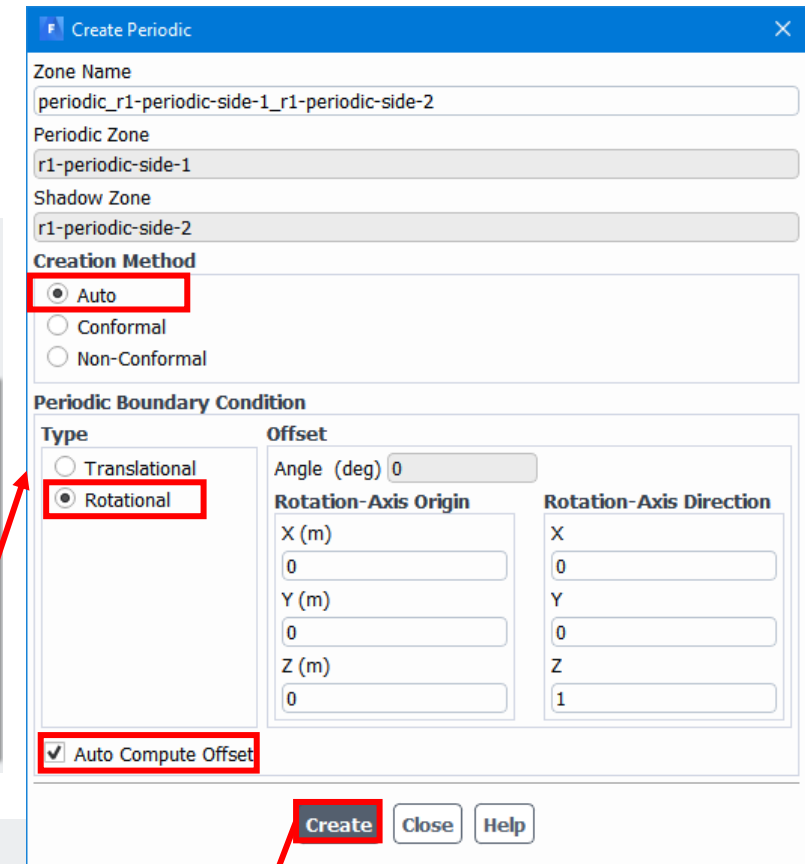
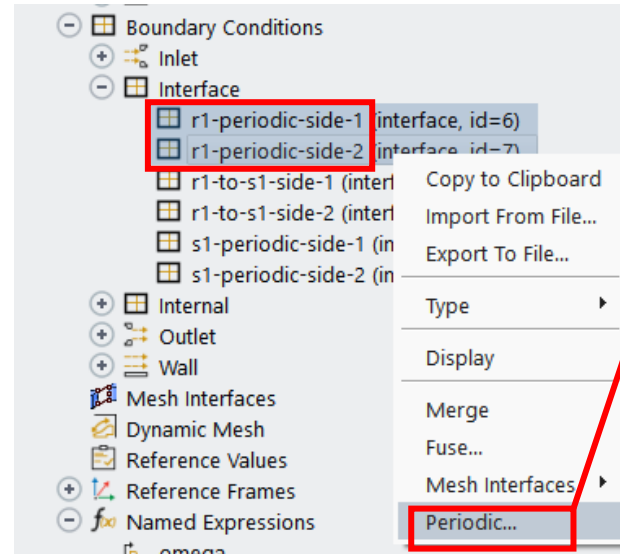
Physics: Cell Zone Conditions (2)

- Continuing from previous slide...
- The settings of cell zone *r1* should look as in the right-top image
- Click *Apply*
- The rotational speed has been set via an expression $\omega = 2880 \text{ [rev/min]}$
 - Sign verification: If you place your right thumb to point as the positive z-axis, your fingers are curling (in this case) to the same direction with the rotation direction of the impeller. Therefore, the Rotational Velocity was set to a positive number
- Expressions are used for a consistent setup and may also be used for the calculation of key targeted quantities
 - ω will be later used, in conjunction with a Force report for the moment, for creating a report for the rotor power



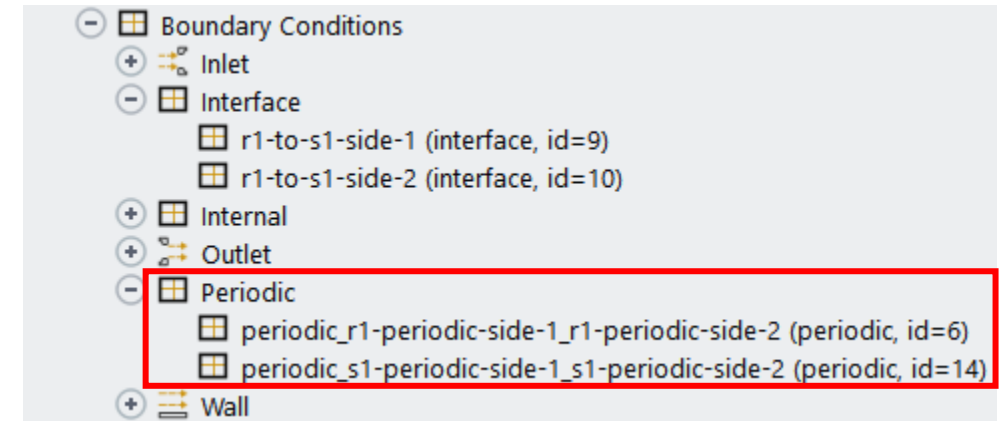
Create Rotational Periodic Zones

- In the *Boundary Conditions* of the *Outline View* expand the *Interface* branch
- Select *r1-periodic-side-1* and *r1-periodic-side-2*
 - Use *Ctrl + RMB* for multiple selections
- *RMB* > *Periodic*
- Select *Rotational* as *Type* and leave all the rest to default values
 - Note that the *Rotation-Axis Direction* is automatically set to the one defined in the cell-zone frame motion
 - It is important to always first define the axis of rotation in the cell-zone conditions before creating the Rotational Periodic boundaries
 - Look for *Console* message:
Zone 7 deleted
Created a conformal periodic boundary.
- The periodic pair is placed under the newly created *Periodic* branch in the *Outline*



Create Rotational Periodic Zones (2)

- In the same way Create Rotational Periodic Zones for *s1-periodic-side-1* & *s1-periodic-side-2*
- Do a mech check
 - Remember, always do a mesh check after creating rotational periodic boundaries
 - This will give you a summary of all Periodic zones including the rotation angle and the axis of rotation
 - Periodic zone 6 corresponds to the rotor and its rotation angle was computed to 12 degrees = 360 degrees / 30 blades (the rotor has 30 blades)
 - Periodic zone 14 corresponds to the stator and its rotation angle was computed to 18 degrees = 360 degrees / 20 blades (the stator has 20 blades)
 - At this stage we can also see a **Warning about an unassigned zone for interface 10**. This will be corrected later by defining the rotor/stator GTI interface



Console

```
WARNING: Unassigned interface zone detected for interface 10.....
Periodic zone   6: average rotation angle (deg) = 12.000 (12.000 to 12.000)
                  stored zone rotation angle (deg)   = 12.000
                  stored axis   , (0.000000e+00, 0.000000e+00, 1.000000e+00)
                  stored origin, (0.000000e+00, 0.000000e+00, 0.000000e+00)

Periodic zone  14: average rotation angle (deg) = 18.000 (18.000 to 18.000)
                  stored zone rotation angle (deg)   = 18.000
                  stored axis   , (0.000000e+00, 0.000000e+00, 1.000000e+00)
                  stored origin, (0.000000e+00, 0.000000e+00, 0.000000e+00)

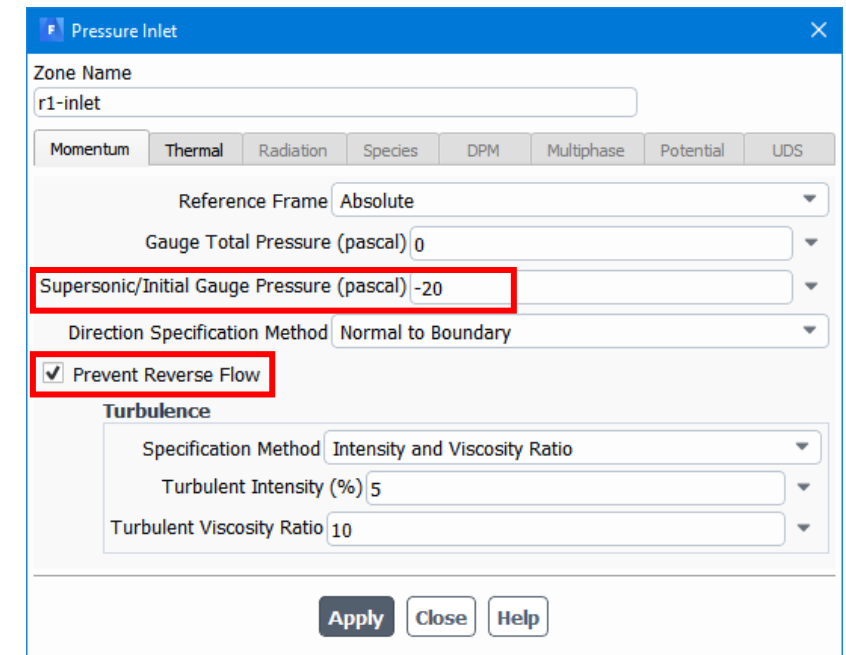
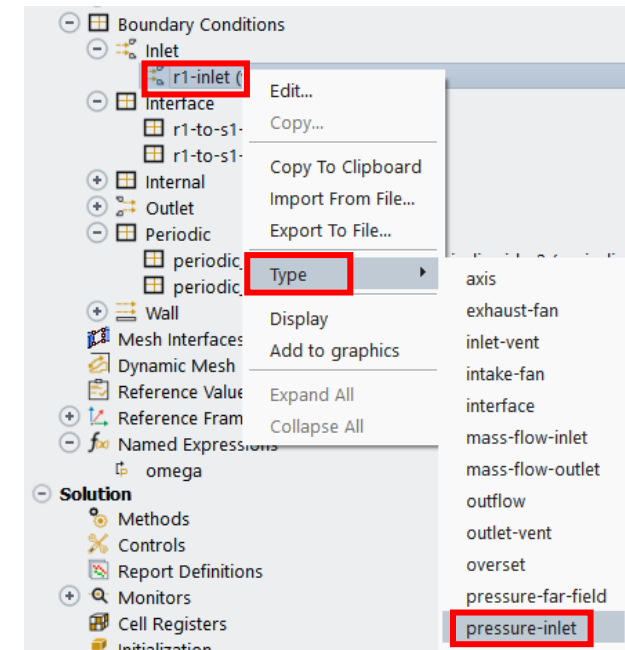
Done.

WARNING: Mesh check failed.

To get more detailed information about the mesh check failure
increase the mesh check verbosity via the TUI command
/mesh/check-verbosity.
```

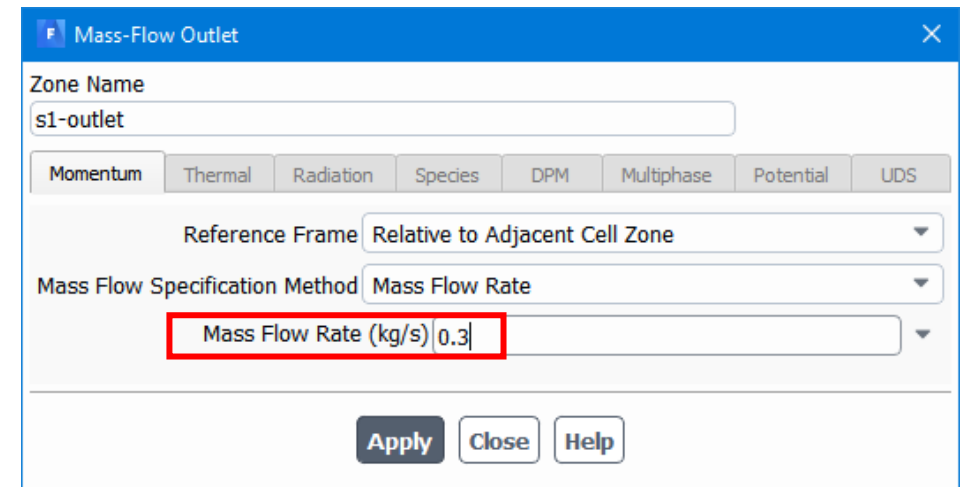
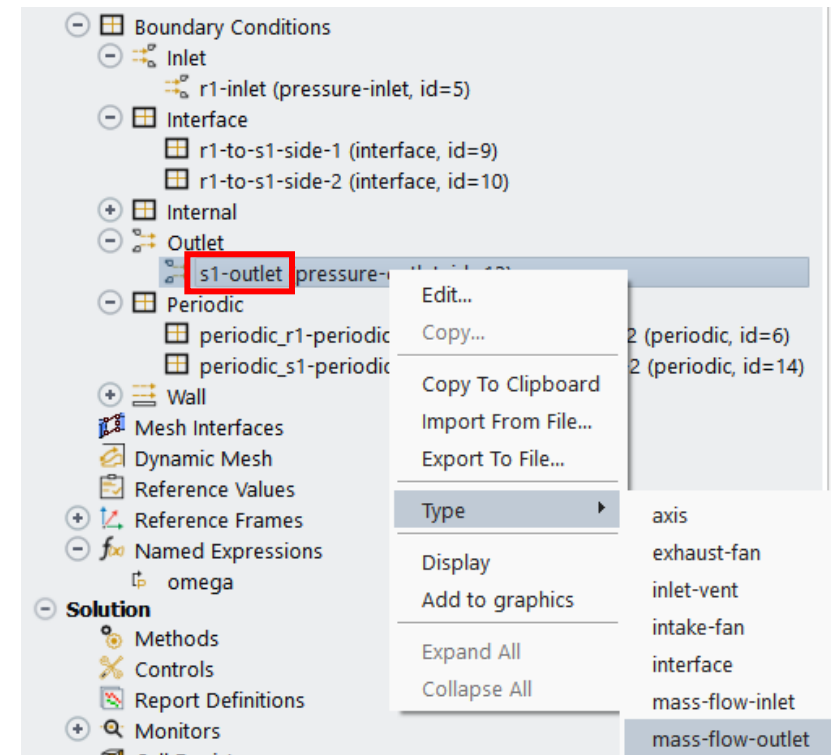
Boundary Conditions: Inlet

- Set the boundary conditions for *r1-inlet*
 - *RMB* on *r1-inlet* and set *Type* to *pressure-inlet*
 - Check *Prevent Reverse Flow*
 - Set a *Gauge Total Pressure* of 0 (pascal) at the inlet *
 - Set a *Supersonic/Initial Gauge Pressure* of -20 (pascal)
 - *Initial Gauge Pressure* is set a few (pascal) lower than the *Gauge Total Pressure*. This will help in the flow field initialization (see slide 25)
 - Accept all remaining defaults in the Momentum tab
 - In the *Thermal* tab, set a *Total Temperature* of 288 (k) (not shown), click *Apply* then *Close*
- * Note that the *Operating Pressure* was set to 100,000 (Pa) for this case. For this reason, the *Gauge Total Pressure* is set to 0 (Pa)



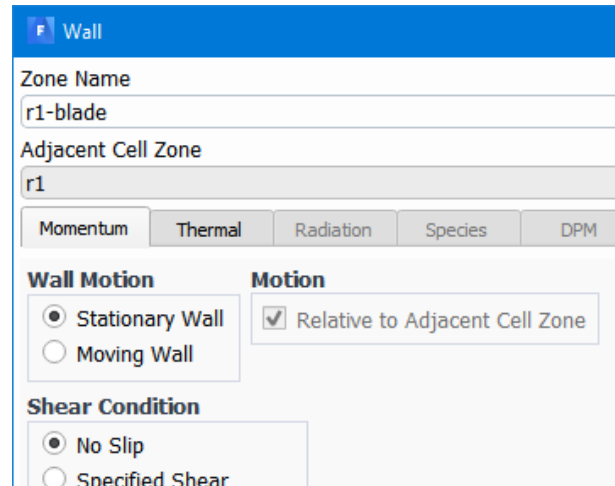
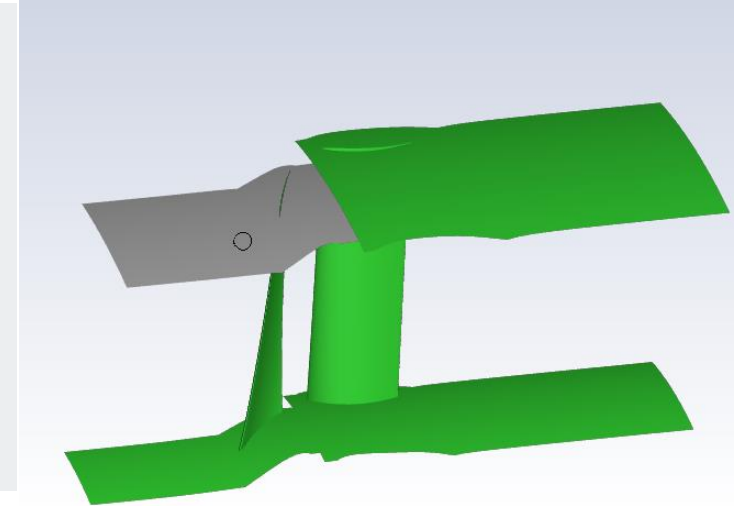
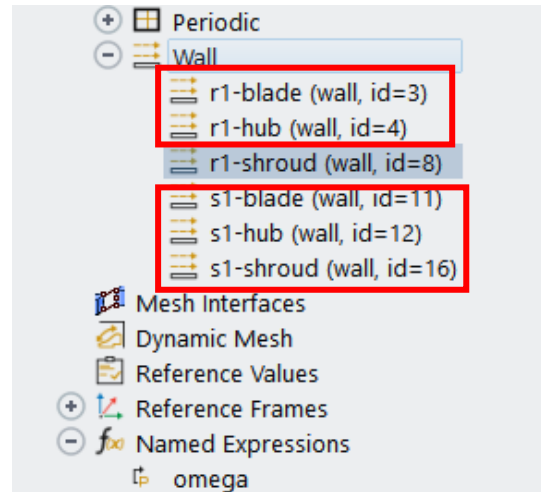
Boundary Conditions: Outlet

- Set the boundary conditions for *s1-outlet*
 - RMB on *s1-outlet* and set *Type* to *mass-flow-outlet*
 - Set the *Mass Flow Rate* = $0.3(\text{kg/s})$
 - Click *Apply*



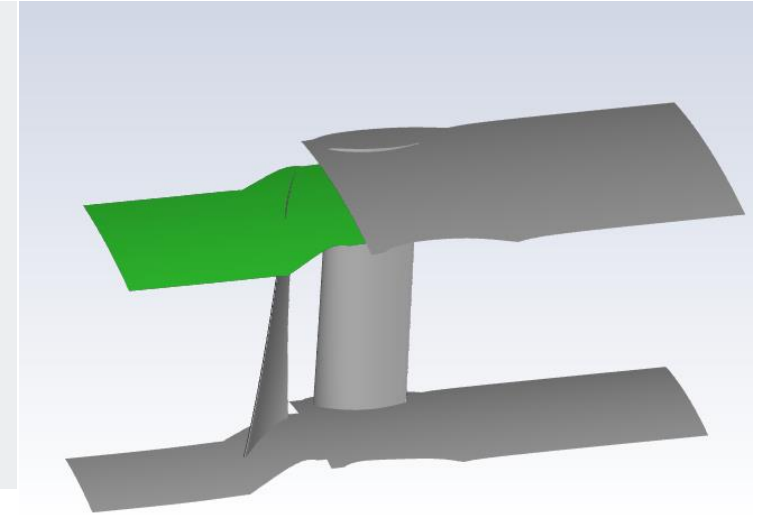
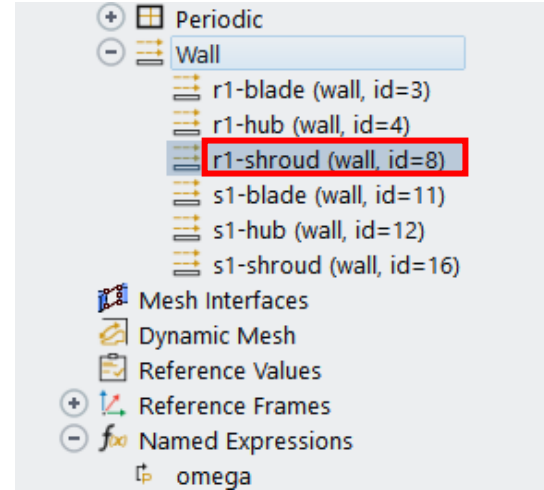
Boundary Conditions: Automatically Set Walls

- From the remaining zones under *Wall* in the *Outline View*, all boundaries marked with red boxes correspond to walls which can be left to the default wall boundary condition setting:
 - *Stationary Wall*
 - *Relative to Adjacent Cell Zone*
 - *No Slip*



Boundary Conditions: Rotor Shroud Wall

- The rotor shroud wall *r1-shroud*, belongs to a rotating cell-zone but is stationary in the absolute frame
 - Double click on *r1-shroud*
 - Do all settings shown on the right and click *Apply*



Wall

Zone Name
r1-shroud

Adjacent Cell Zone
r1

Momentum Thermal Radiation Species DPM Multiphase UDS Potential Structure

Wall Motion

☐ Stationary Wall
☒ Moving Wall

Motion

☐ Relative to Adjacent Cell Zone
☒ Absolute

Speed (rad/s) 0

Rotation-Axis Origin

X (m) 0
Y (m) 0
Z (m) 0

Rotation-Axis Direction

X 0
Y 0
Z 1

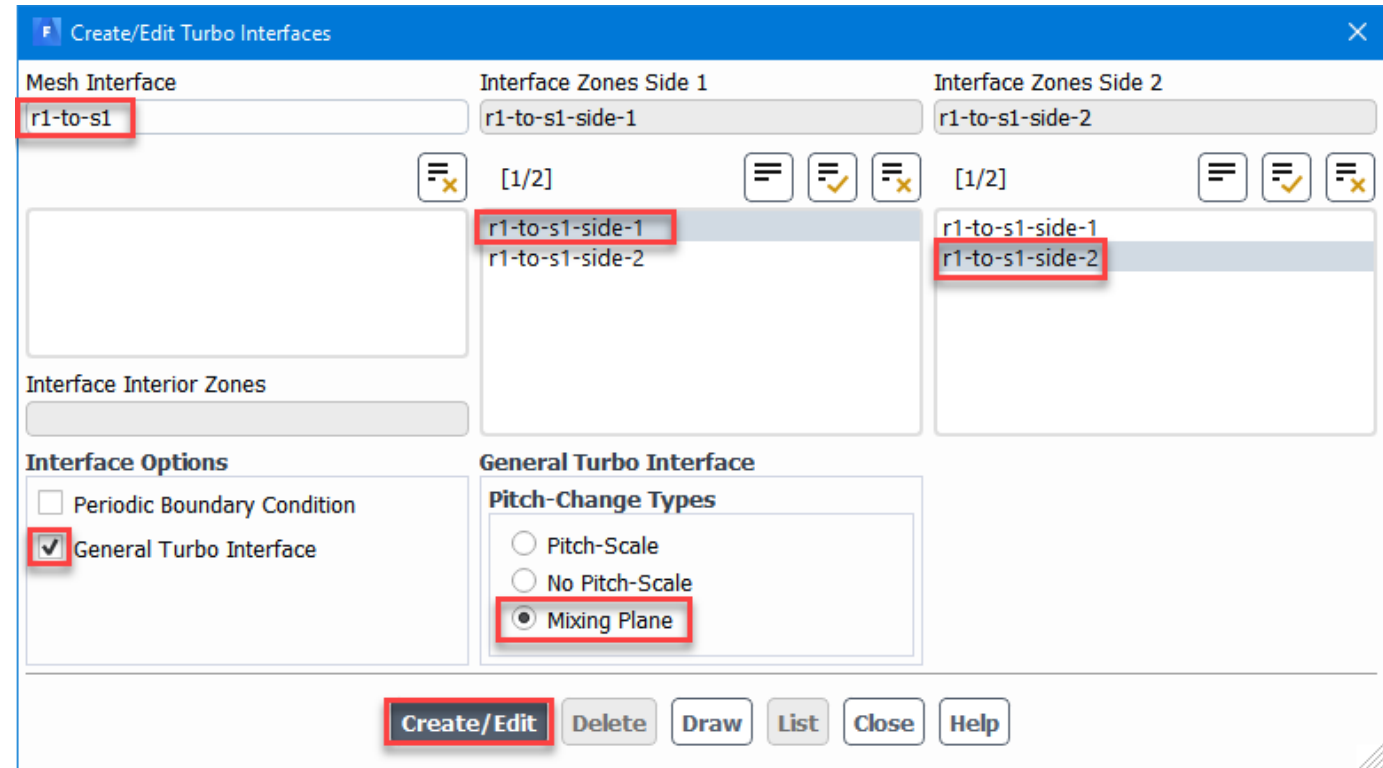
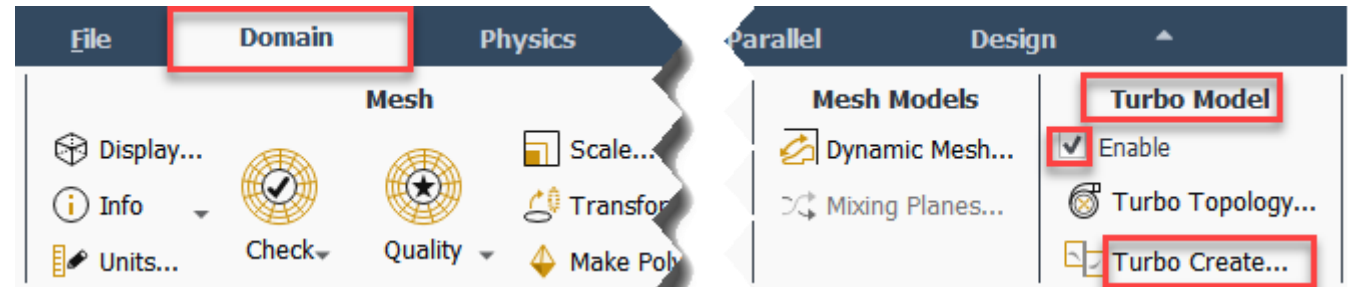
Shear Condition

☒ No Slip
☐ Specified Shear
☐ Specularity Coefficient
☐ Marangoni Stress

Such walls are currently set as Moving Walls with Rotational Motion relative to the Absolute frame with a zero Rotational speed

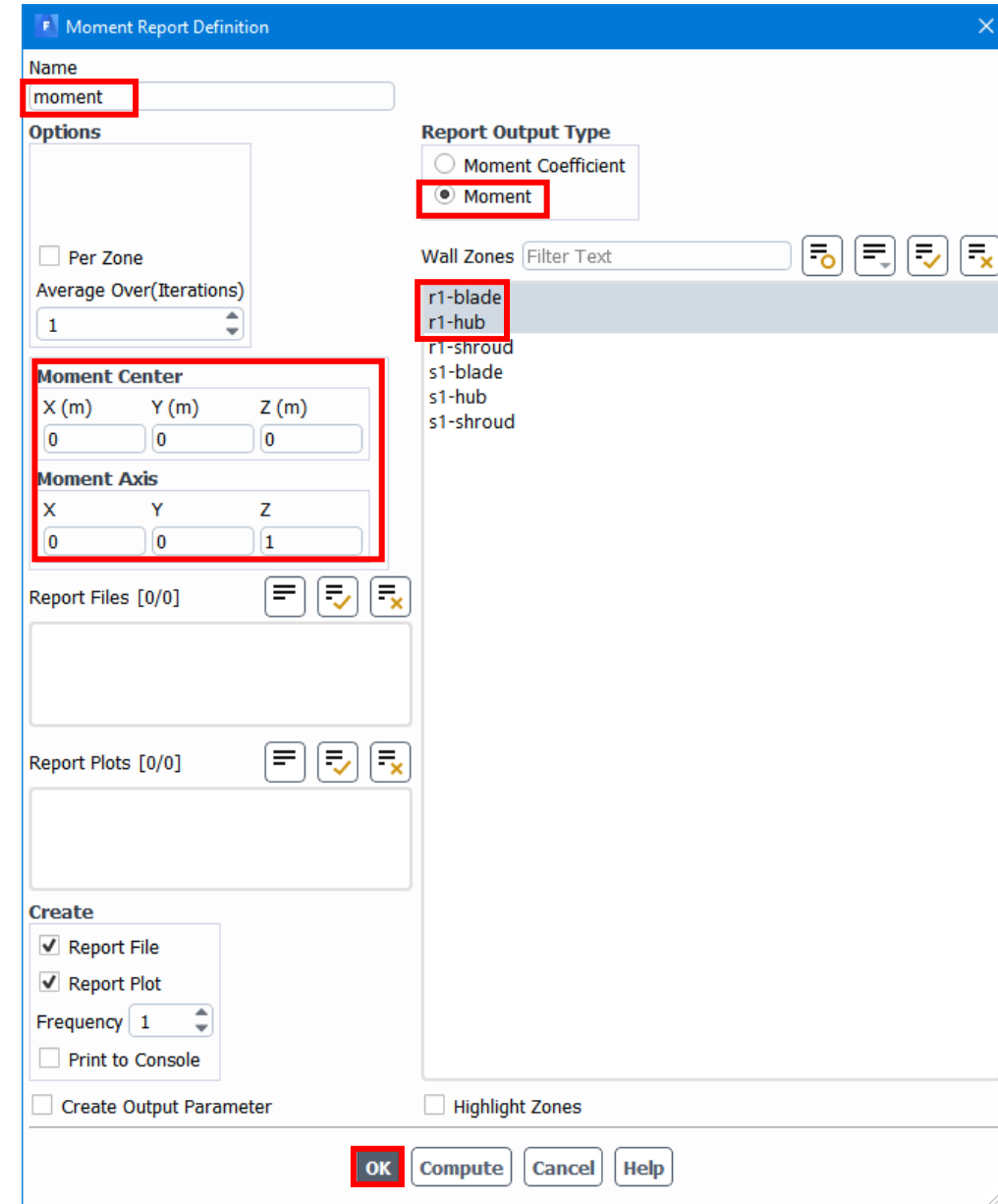
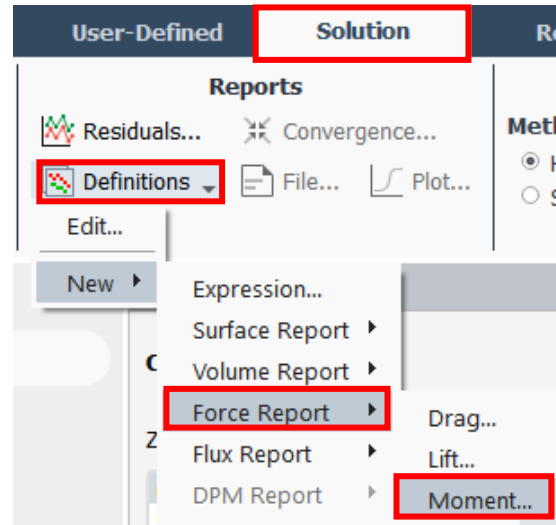
Define a Mixing Plane Interface

- Having concluded with all rotational periodic zones and basic boundary conditions, you will now define a *Mixing Plane General Turbo Interface*
 - In the *Domain* tab under *Turbo Model* group check *Enable* and click *Turbo Create...*
 - In the *Create/Edit Turbo Interfaces* panel:
 - Set mesh interface name to *r1-to-s1*
 - Check *General Turbo Interface*
 - Select the two sides of the interface
 - Select *Mixing Plane*
 - Click *Create/Edit*
 - Ignore any error message in the console after clicking *Create/Edit*
 - Do a mesh check and make sure that no error message appears in the console



Solution: Moment Report Definition

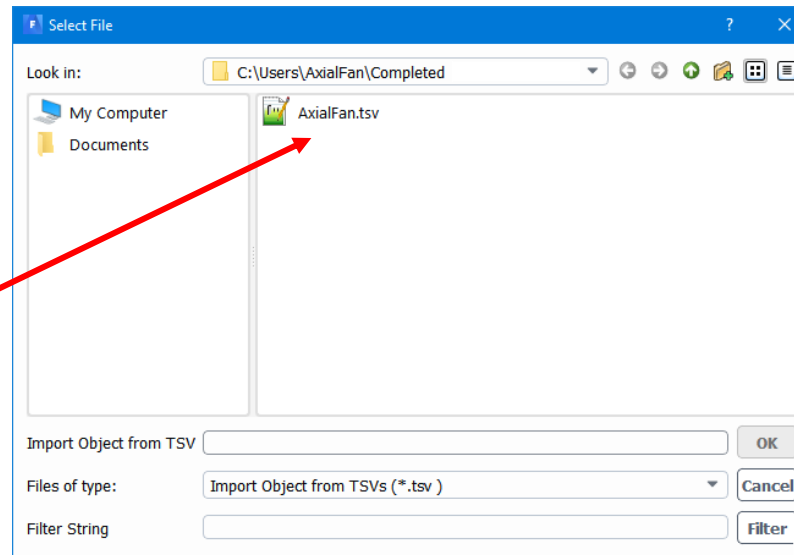
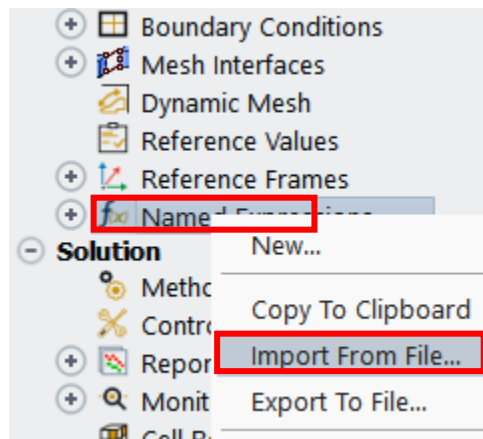
- In the *Solution* tab create a new *Force Report Definition* for the *Moment* about the *z-axis*, with the following settings:
 - *Name* = *moment*
 - *Report Output Type* = *Moment*
 - *Boundaries* = *r1-blade*, *r1-hub*
 - *Report File* = checked
 - *Report Plot* = checked
 - *Moment Axis* → keep the default corresponding to the z-axis)
 - Click *OK*



New Method for Report Definitions Using Named Expressions

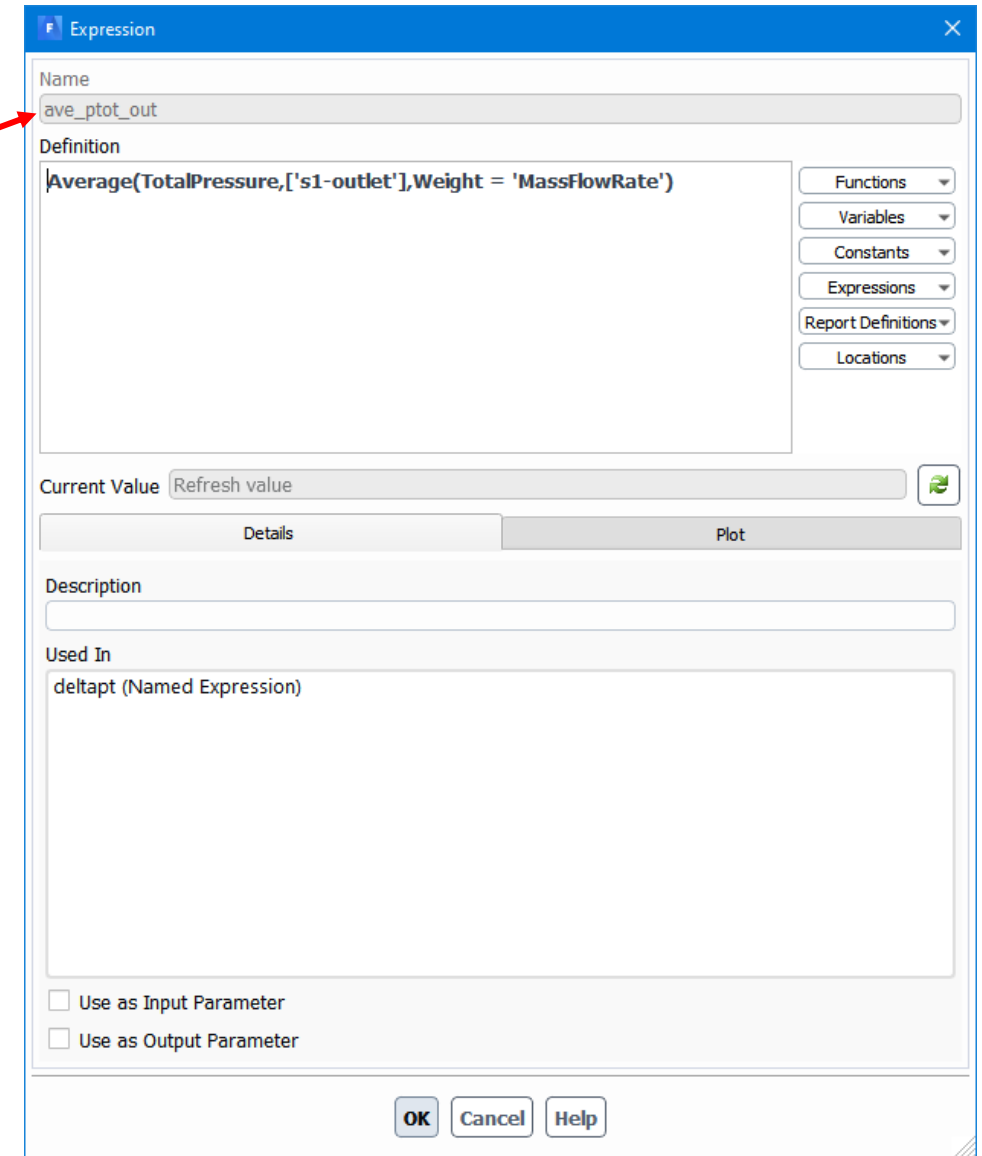
- You will now use a new method for creating Report Definitions and Output Parameters, based on Named Expressions
 - Named Expressions are introduced in the Fluent Getting Started course in the “Setting up Physics” lecture
 - In this workshop for convenience, a file *AxialFan.tsv* is provided with the workshop inputs, containing the syntax for 3 named Expressions:
- You can import this file into the *Named Expressions* branch of the *Outline* using *RMB>Import From File...*
- The 3 *Named Expressions* highlighted by a red box in the bottom-right image are created

| name | definition | description | input-parameter | output-parameter |
|----------------|--|-------------|-----------------|------------------|
| "ave_ptot_in" | "Average(TotalPressure,['r1-inlet'],Weight = 'MassFlowRate')" | | | |
| "ave_ptot_out" | "Average(TotalPressure,['s1-outlet'],Weight = 'MassFlowRate')" | | | |
| "deltapt" | "ave_ptot_out-ave_ptot_in" | | | |



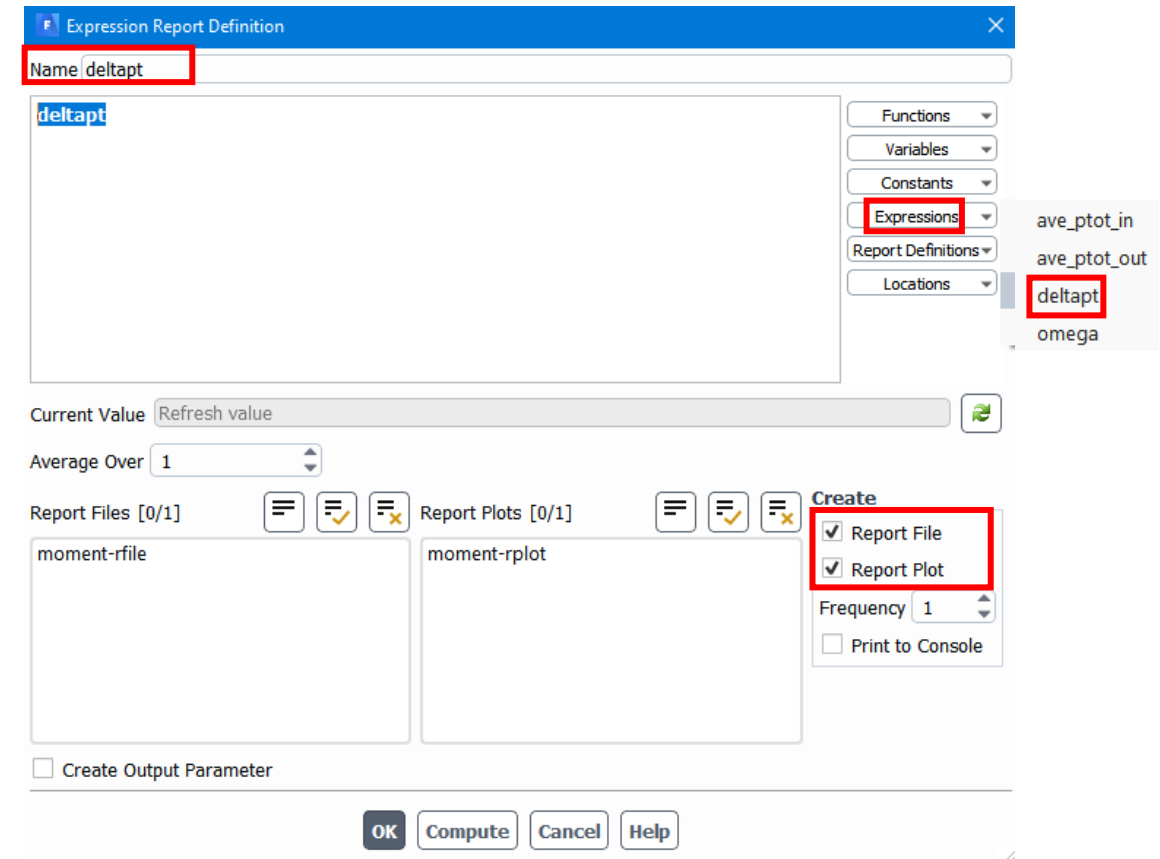
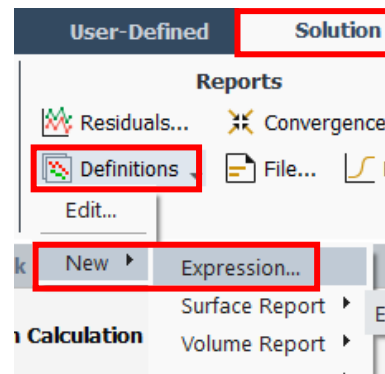
New Method for Report Definitions Using Named Expressions (2)

- Double click on any of the new *Named Expressions* for examining its definition in the *Expression* editor



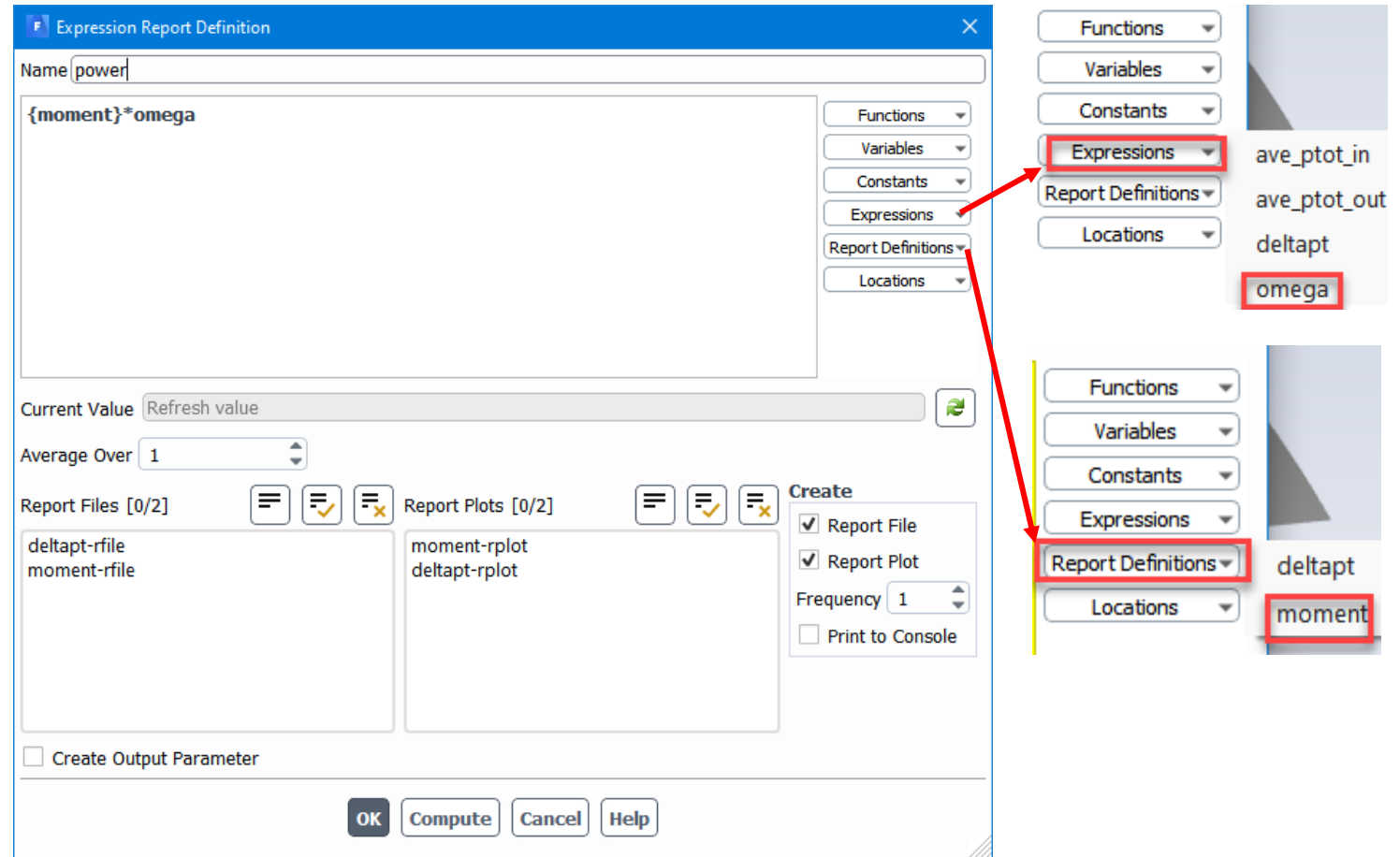
Solution: Create Report Definitions Using Named Expressions

- In the *Solution* tab click on *Definitions* in the *Reports* section and choose *New > Expression...*
Enter the following in the definition panel and click *OK*:
 - *Name* = *deltapt*
 - *Expressions* > *deltapt*



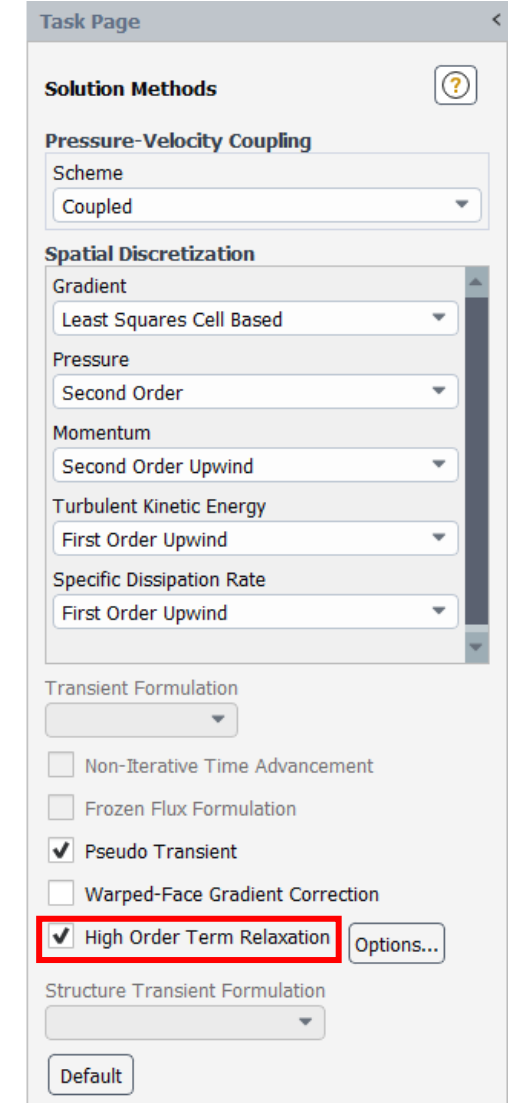
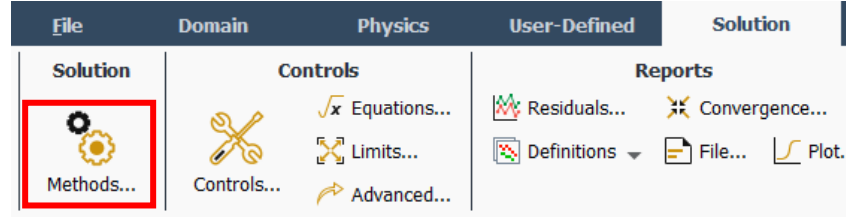
Solution: Create Report Definition for the Fan Power

- In the *Solution* tab click on *Definitions* in the *Reports* section and choose *New >Expression...*
Enter the following in the definition panel and click *OK*:
 - Name = *power*
 - $\{moment\} * \omega$
 - This expression can be typed directly in the expression definition box, or one can use the drop-down lists of existing *Expressions* for *omega* and *Report Definitions* for *moment*



/ Solution: Solution Methods

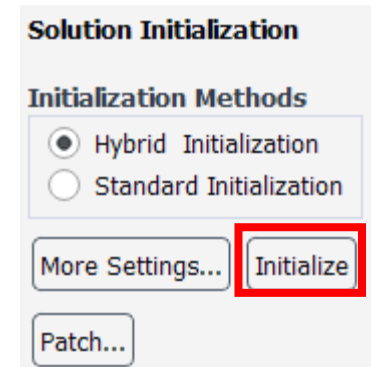
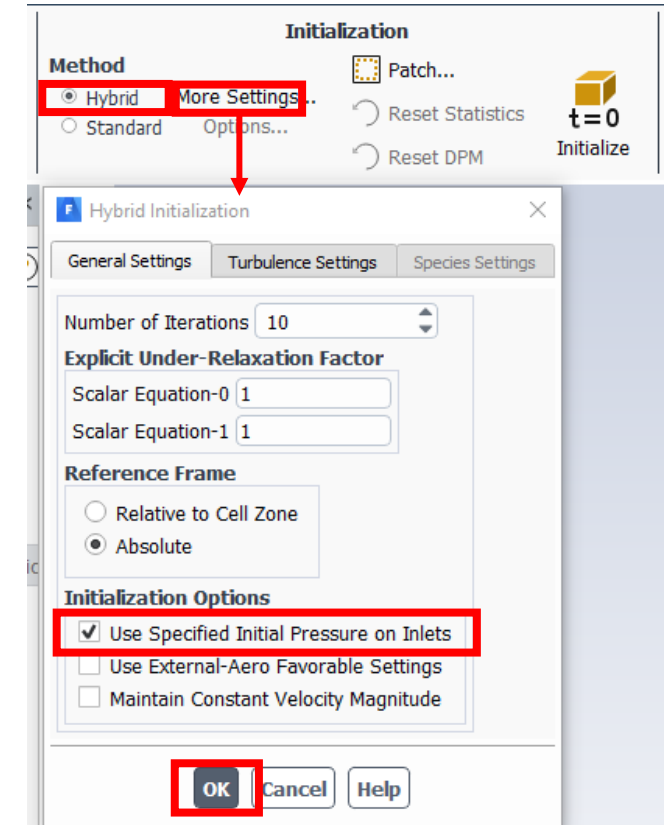
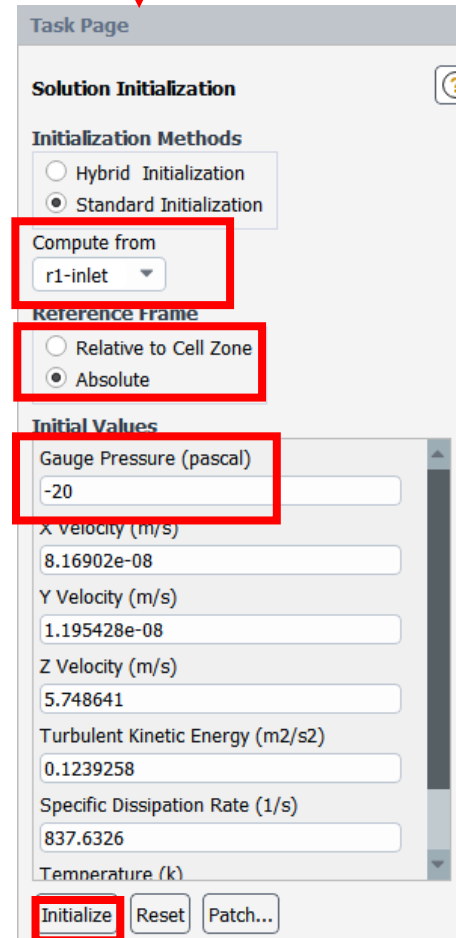
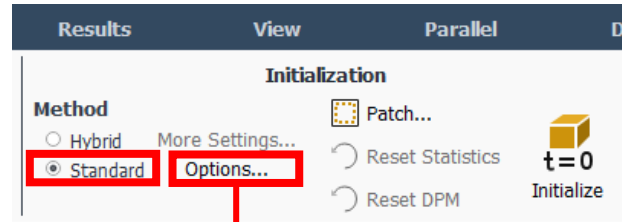
- Always use the default *Coupled “Pseudo-Transient” Solver* for turbomachinery calculations
 - If for any reason the Solution method is set to some *Scheme* other than *Coupled*, click the *Default* button at the bottom of the panel
- Turn on *High Order Term Relaxation* (more stable)



Solution: Initialization (Best Practice Procedure)

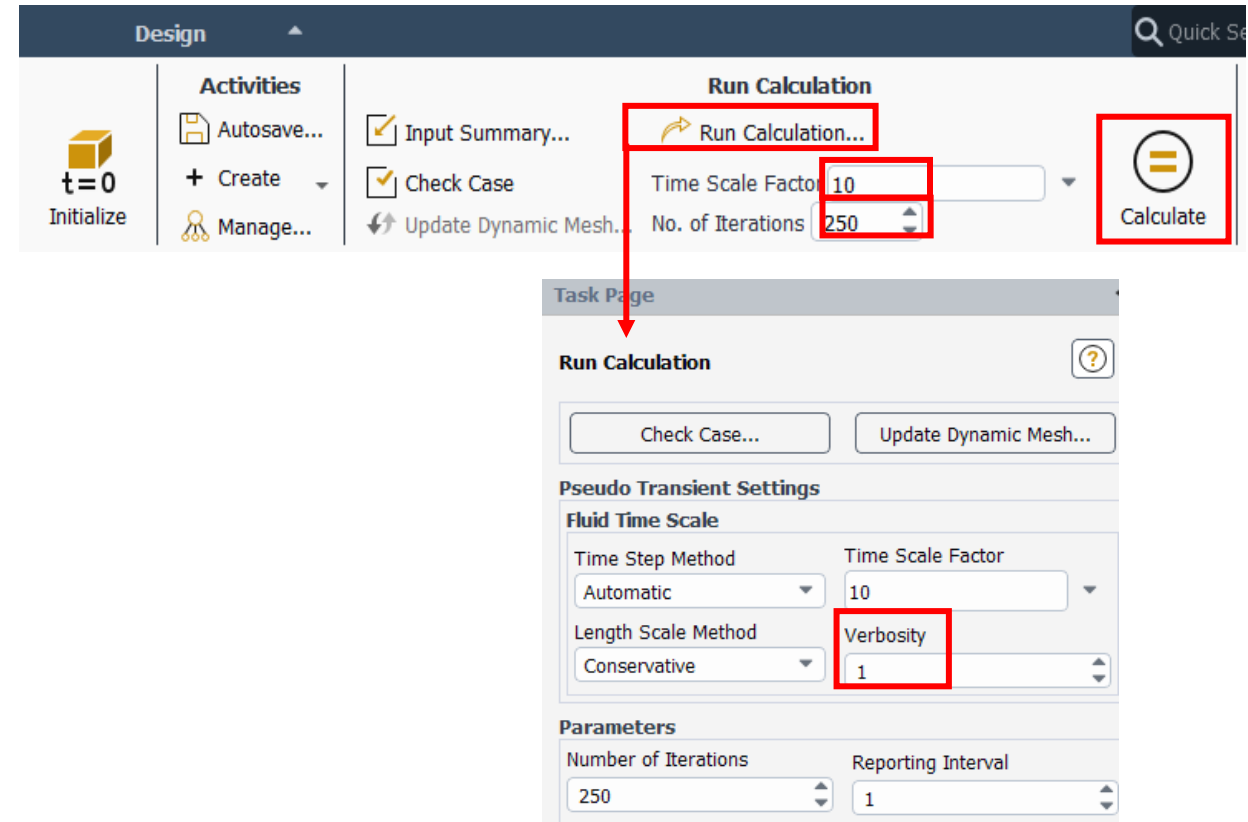
- Perform a *Standard Initialization* using the inlet values
 - Remember we have set the *Initial Gauge Pressure*, a bit lower than the *Gauge Total Pressure* (slide 14)
 - This will ensure a proper k and omega initialization
- Then perform a *Hybrid Initialization*, after having checked the option *Use Specific Initial Pressure on Inlets*
- Save a Fluent .cas and .dat file
 - *File > write > Case & Data...*

Note: FMG initialization is currently not compatible with all General Turbo Interfaces



/ Solution: Run the Solver

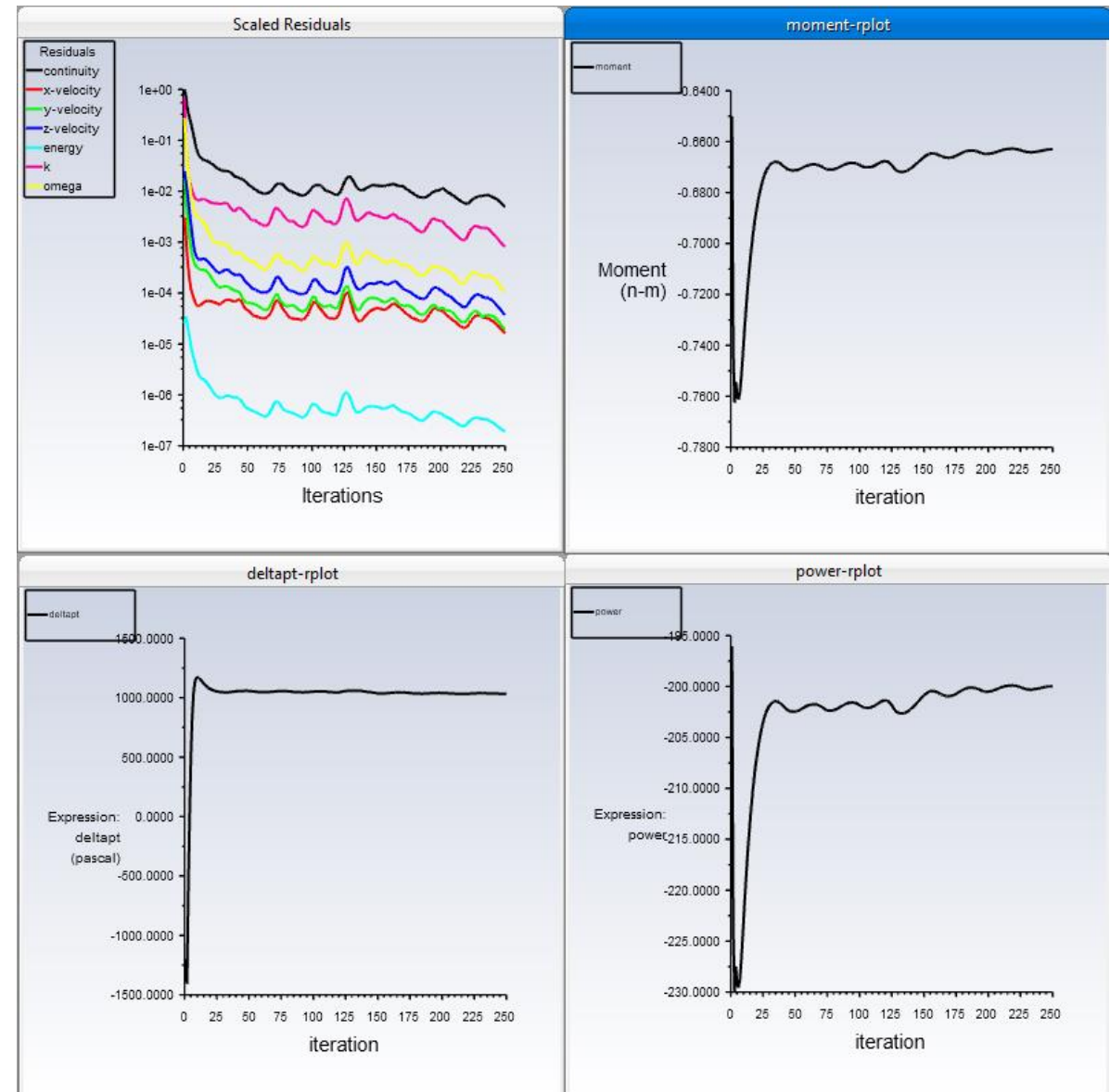
- Click on *Run Calculation...*
 - Set *Verbosity* to 1
 - This will produce a more detailed runtime solver output, including the time step used by the pseudo-transient solver
- Set *Time Scale Factor* to 10
- Set *No. of Iterations* to 250
- Click *Calculate*



/ Solver Convergence

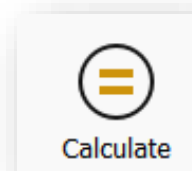
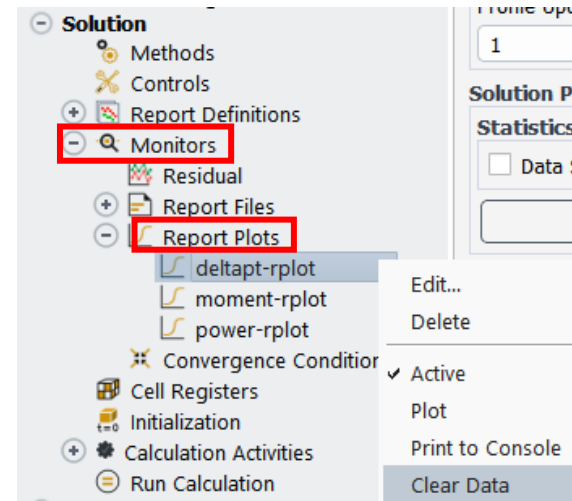
- The case does not converge well to a steady state solution after 250 iterations
 - Residuals and report plots show a bouncy behavior

Note: Residuals and monitor plots may differ between two computers or Ansys releases



/ Check Solver Convergence Further

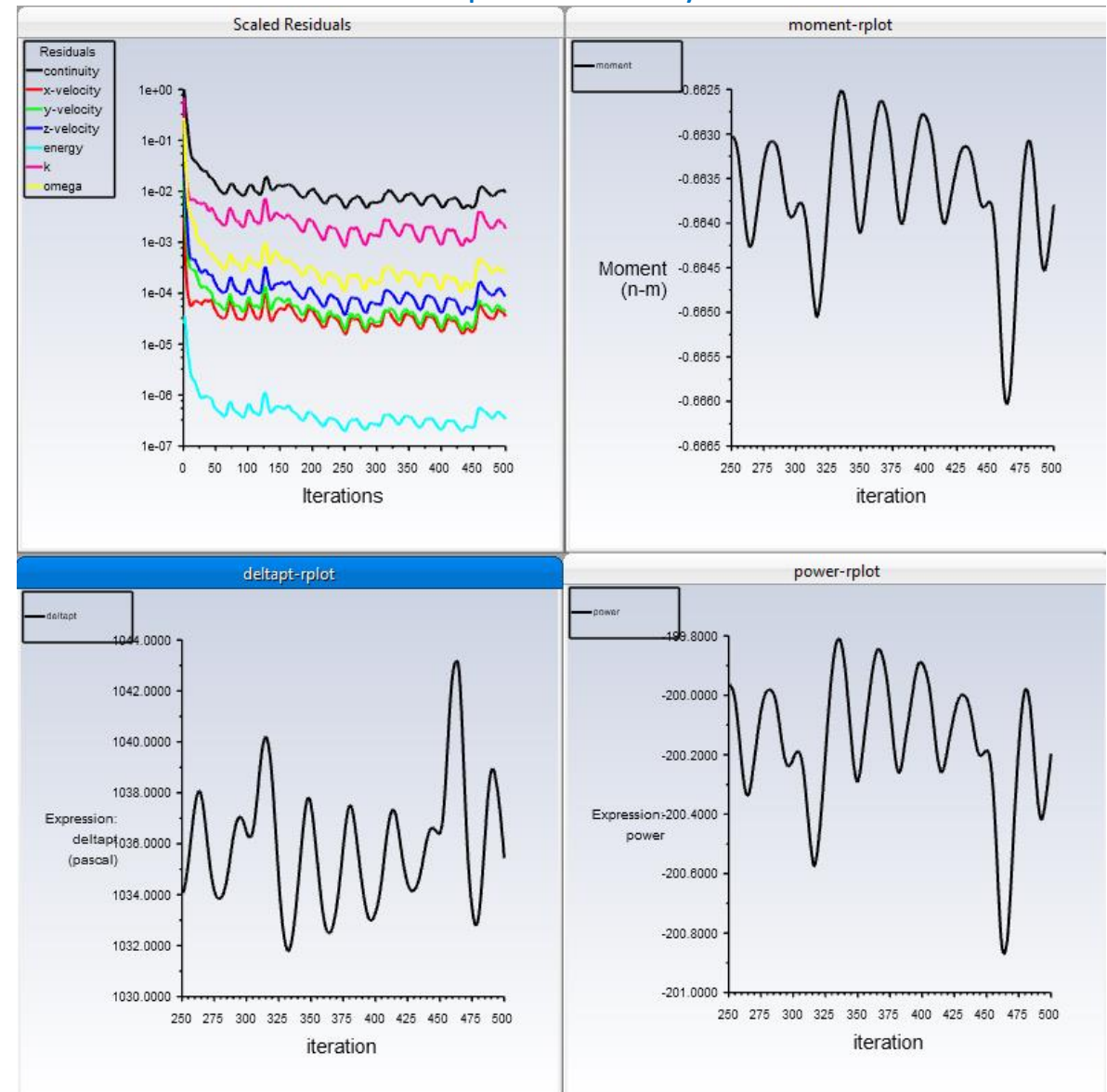
- It is a good idea to do a more detailed check of residuals and monitors.
To do this:
 - Clear all *Report-Plots* data
 - This can be done by expanding the *Monitors* branch in the *Outline* and *RMB* > *Clear Data* for each *Report Plot*
 - This will reset the x- and y-axes limits of the report plots (see next slide)
 - Click *Calculate* in the *Solution* tab in the ribbon, for performing more solver iterations



Solver Convergence

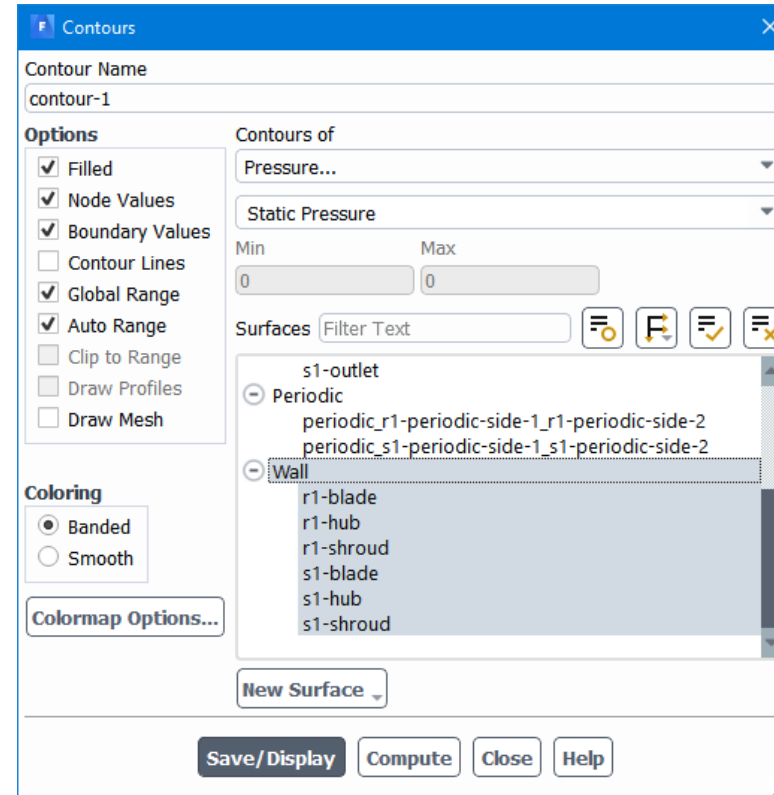
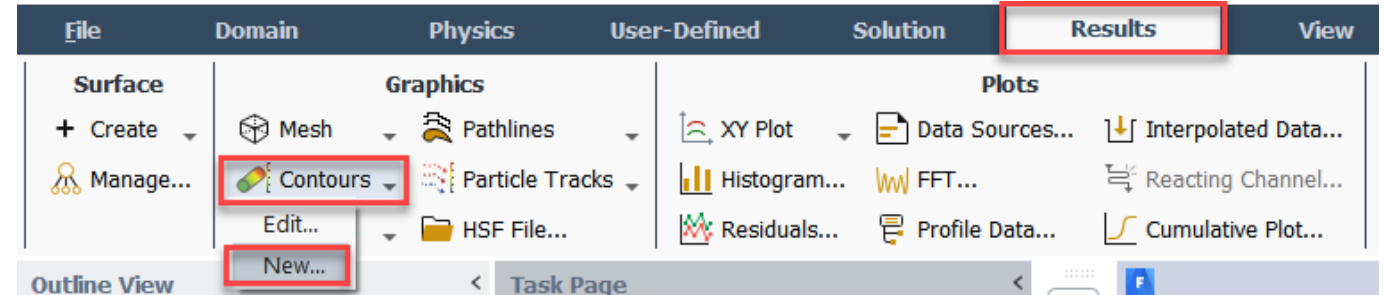
- Residuals and report plots still show a bouncy behavior
 - Continuity residuals oscillate around $1e-2$ and cannot be reduced further
 - Total pressure difference, moment and power monitors oscillate
 - Case is not converged to steady state in the strict sense
 - The targeted quantities of total pressure rise *deltapt* and power consumption *power* seem to be constant within 3 significant figures
 - deltapt* ≈ 1035 (pascal)
 - power* ≈ -200 (watt) per rotor passage
 - Total power consumed = $-200 \times 30 = -6000$ (watt)
 - To check the accuracy of these key quantities, it is a good idea to investigate the case by solving it also as transient. This is done in the next workshop

Note: Residuals and monitor plots may differ between two computers or Ansys releases

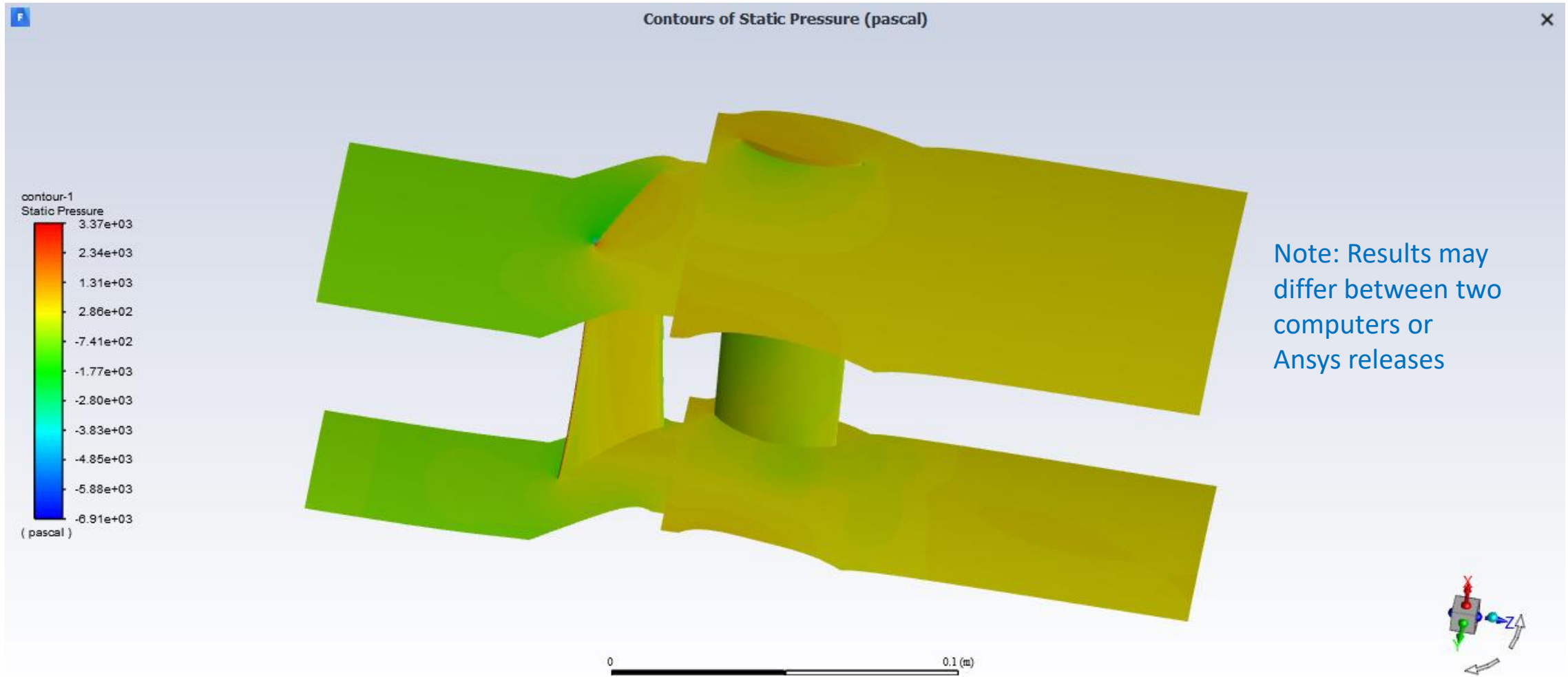


Pressure Contours on the Walls

- Create a *New Contour* plot of *Static Pressure* on all walls

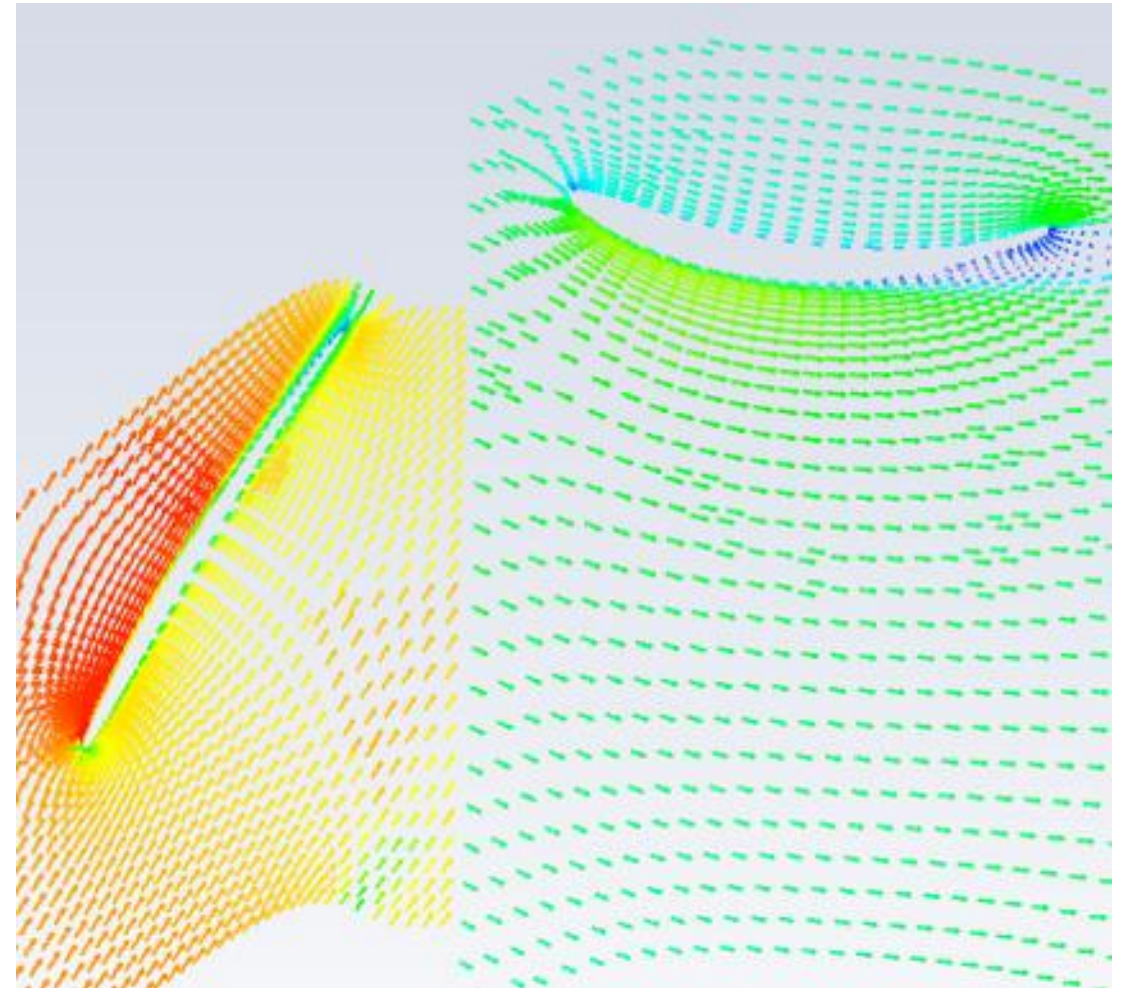


Pressure Contours on the Walls



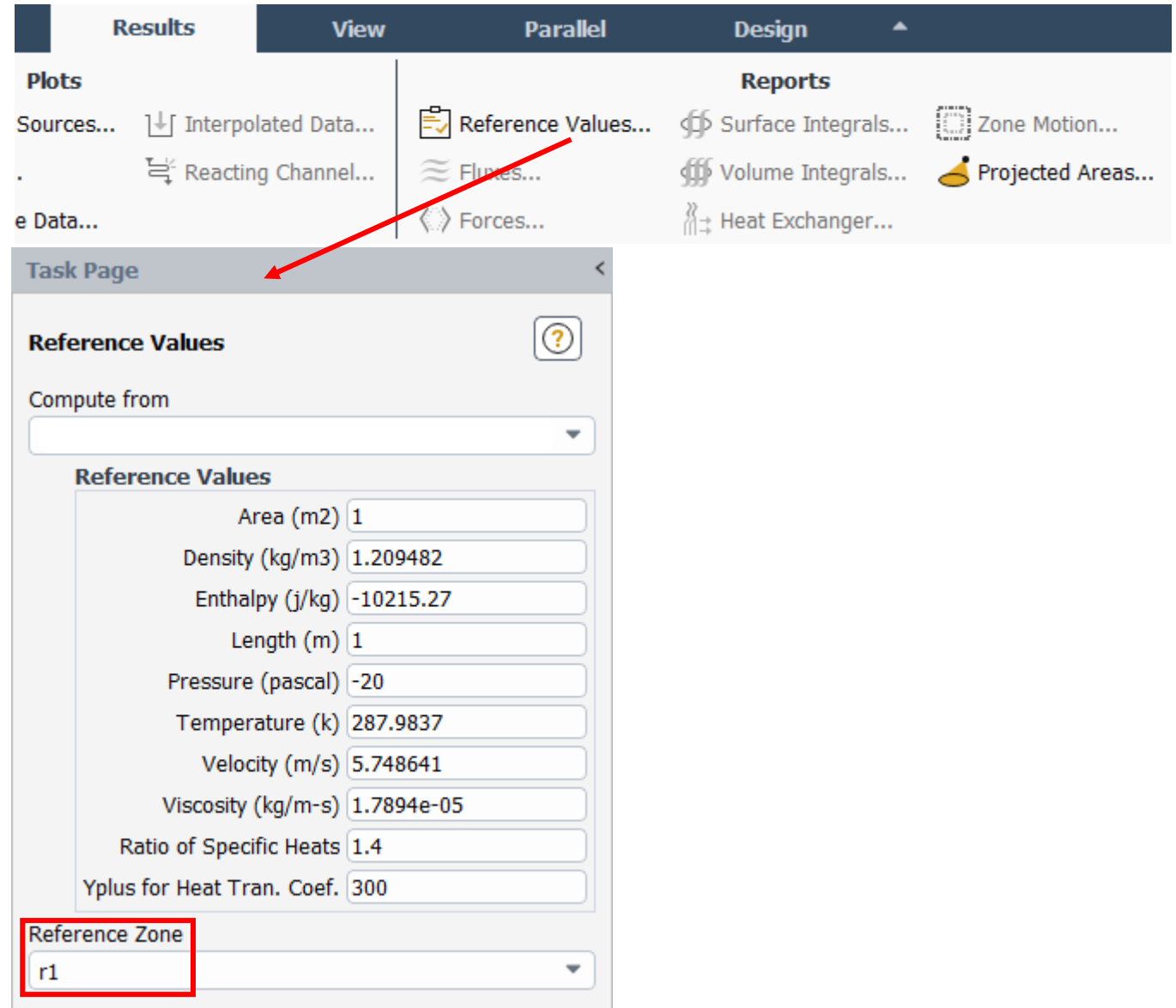
/ Create a Vector Plot on an Iso-surface Constant Radial Coordinate

- You will now create a combined vector plot on an iso-surface of constant radial coordinate at mid-span
- This will include a relative-velocity vector plot for the rotor zone r1 and an absolute-velocity vector plot for the stator zone s1
- For this you will need to:
 - Define the reference zone for relative velocities and
 - Create:
 - 2 Iso-surfaces of constant radial coordinate, one for zone r1 and one for zone s1
 - 2 vector plots, one for zone r1 and one for zone s1
 - 1 scene combining the 2 vector plots



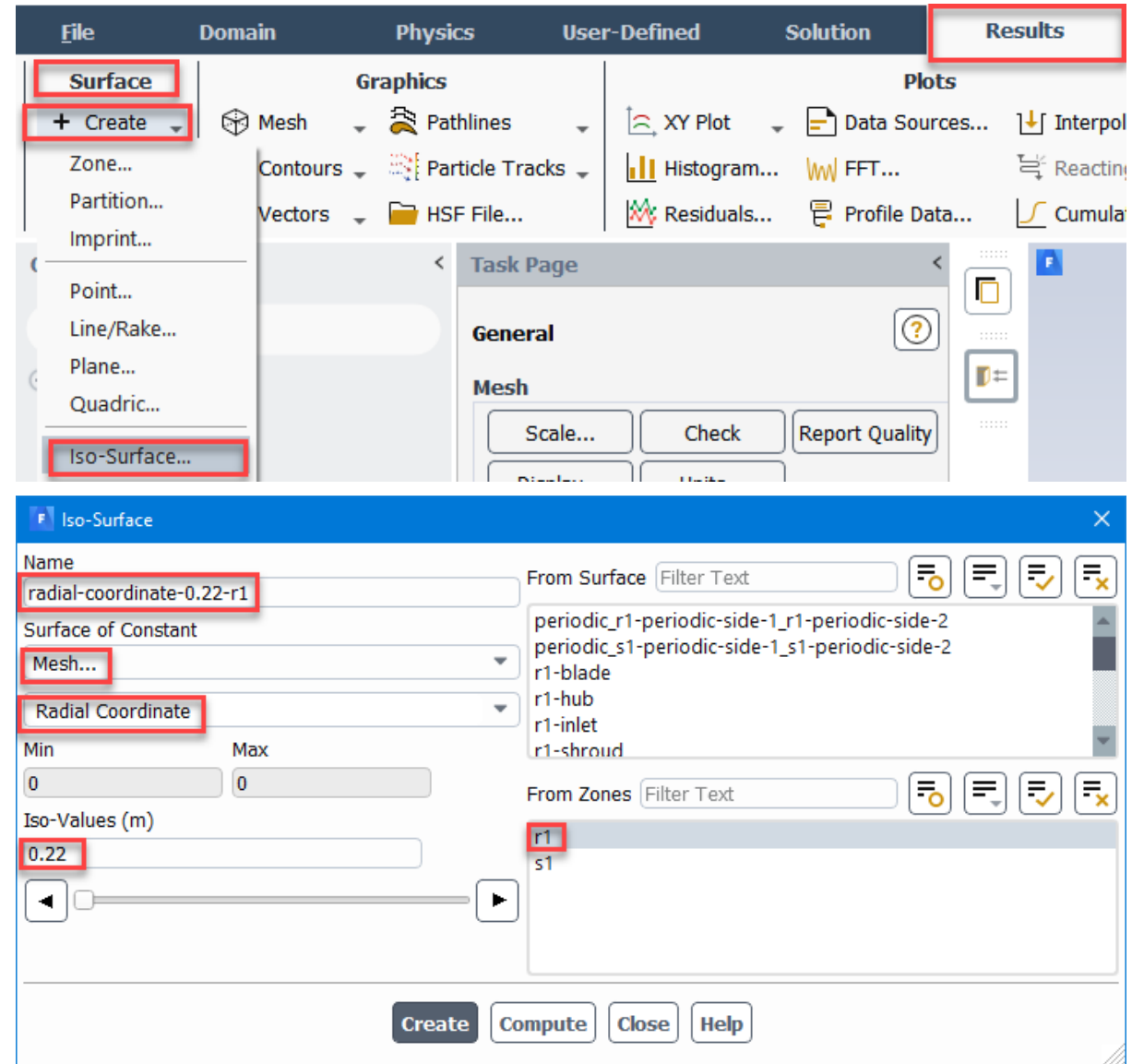
Define Reference Zone for Relative Velocities

- In the *Reference Values* panel set the *Reference Zone* to *r1*
 - In this model, we have one moving and one stationary fluid zone. The *Reference Zone* determines how the relative velocities are computed. See lecture 03 on Post-processing for more details



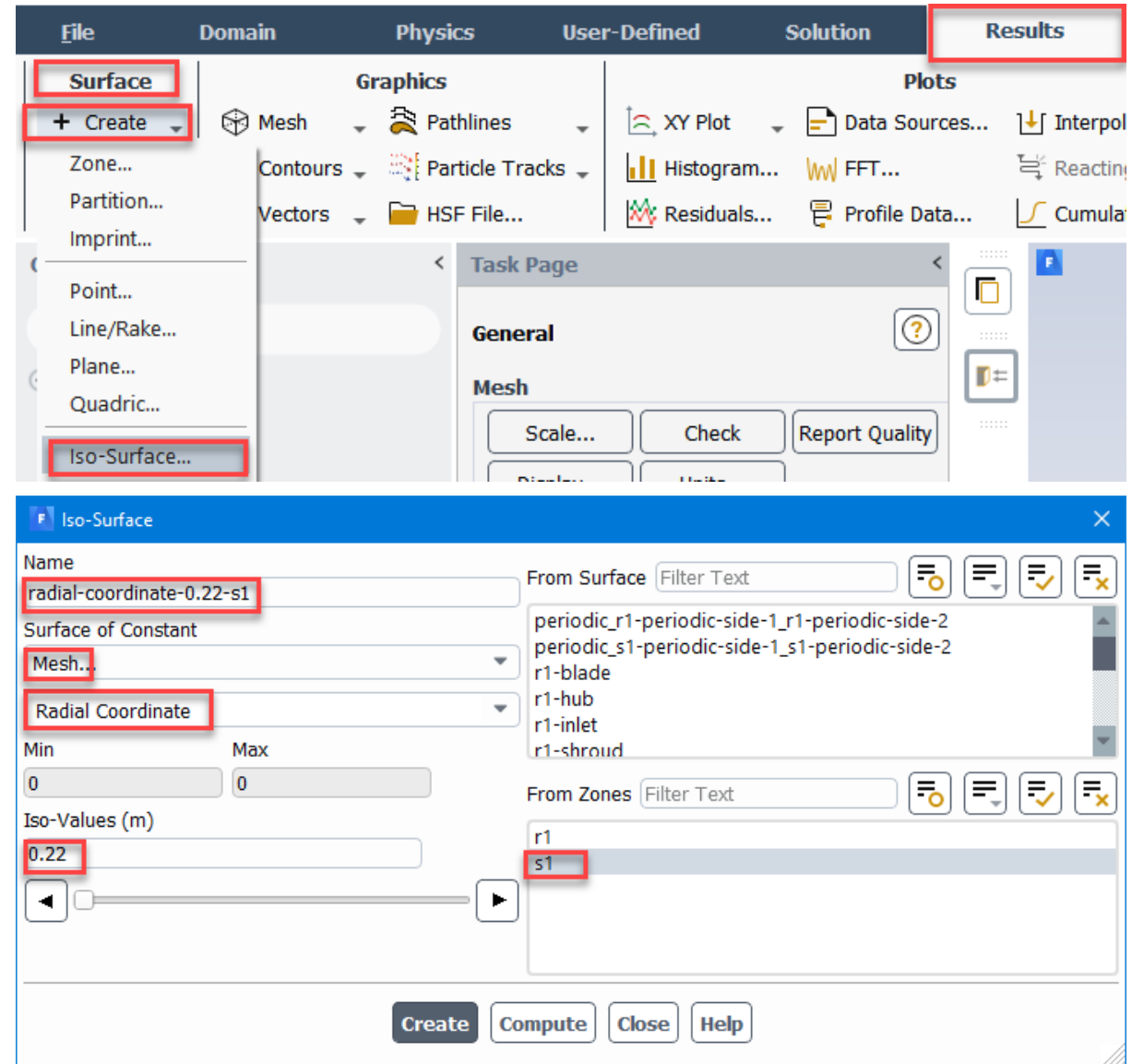
Create a Constant Radial Coordinate Iso-surface for Zone r1

- In the *Results* tab create a Constant Radial Coordinate Iso-surface for Zone r1
 - *Surface of Constant Mesh...*
 - *Radial Coordinate*
 - *Iso-Value = 0.22*
 - This corresponds to a midspan surface. We will use it for creating various vector and contour plots
 - Click *Create*



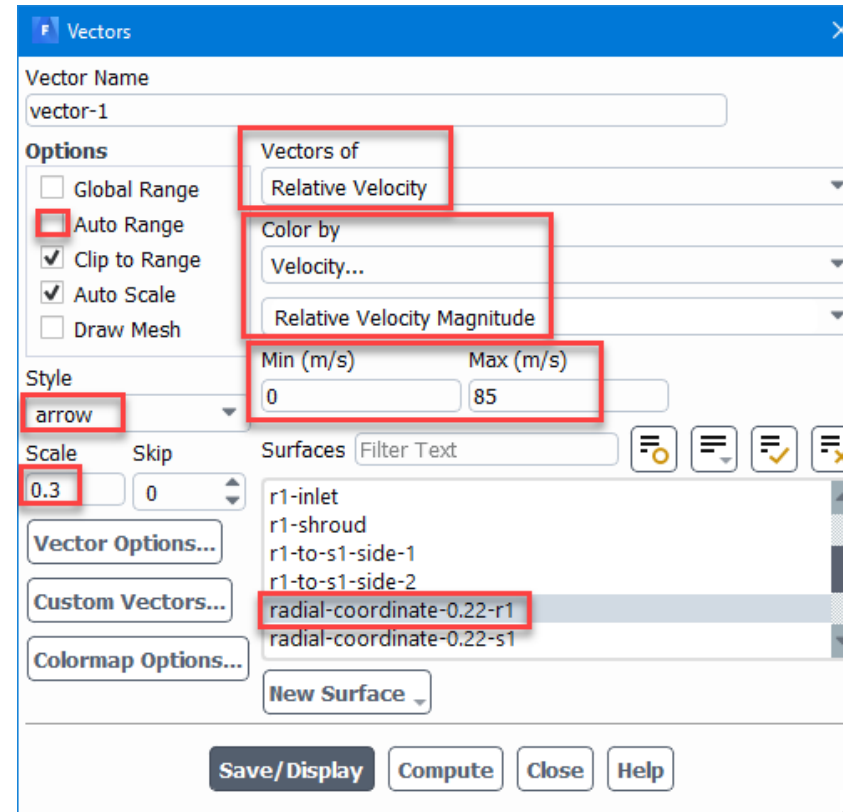
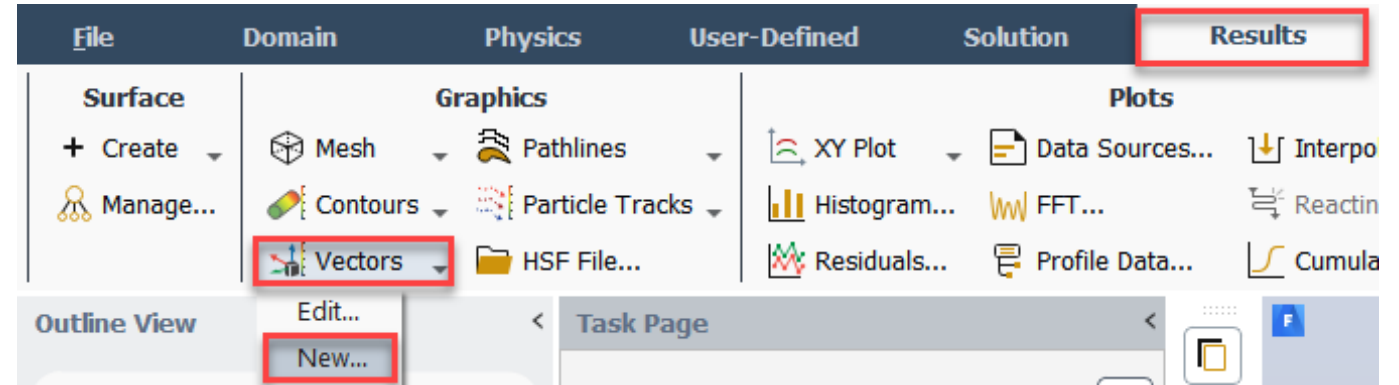
Create a Constant Radial Coordinate Iso-surface for Zone s1

- In the *Results* tab create a Constant Radial Coordinate Iso-surface for Zone s1
 - *Surface of Constant Mesh...*
 - *Radial Coordinate*
 - *Iso-Value = 0.22*
 - This corresponds to a midspan surface. We will use it for creating various vector and contour plots
 - Click *Create*



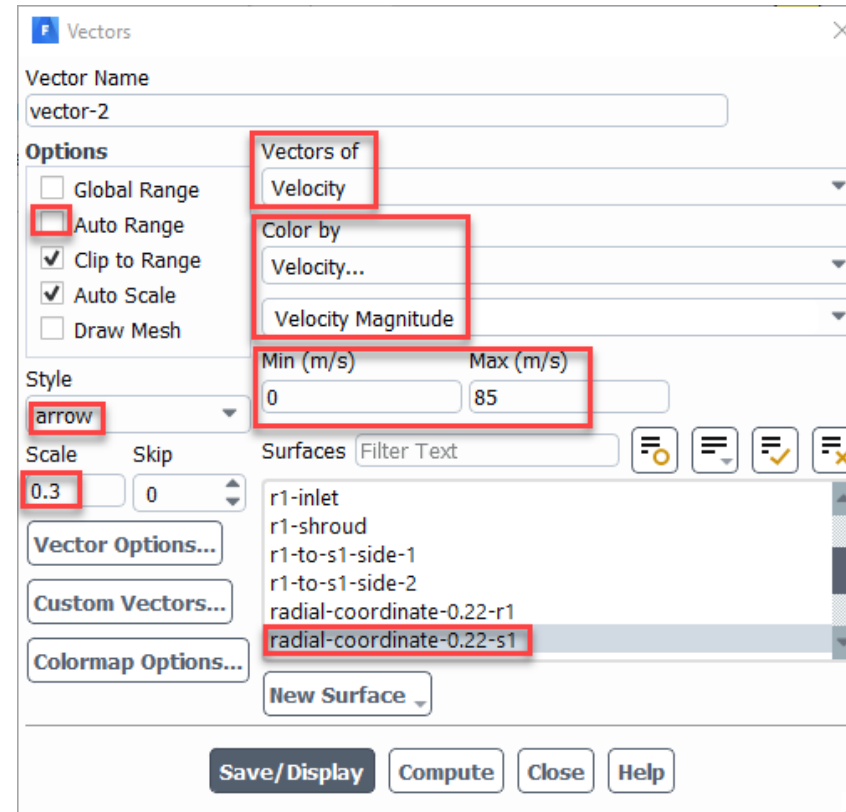
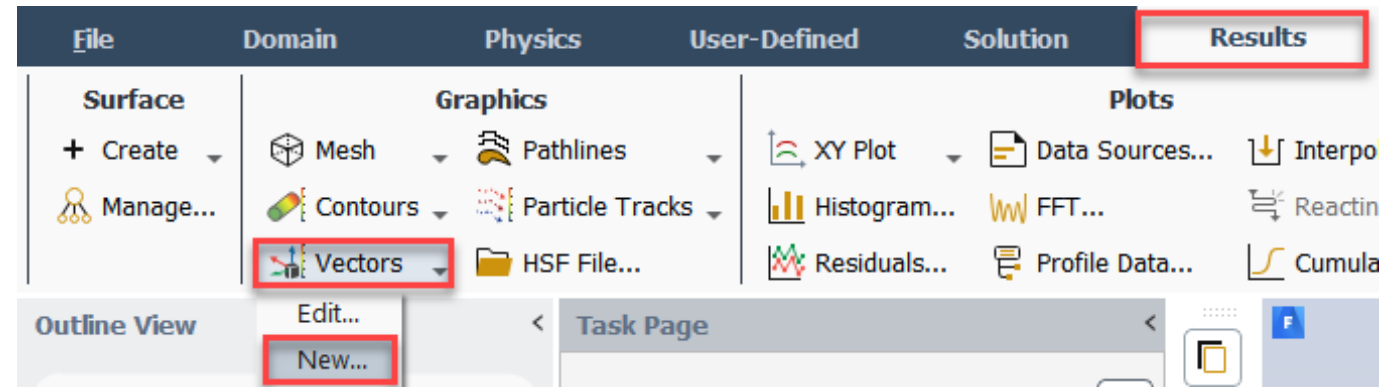
Mid-span Relative Velocity Vector Plot Iso-surface for Zone r1

- Create a *New Vector* plot of *Relative Velocity* on the midspan plane *radial-coordinate-0.22-r1*
- Click *Save/Display*



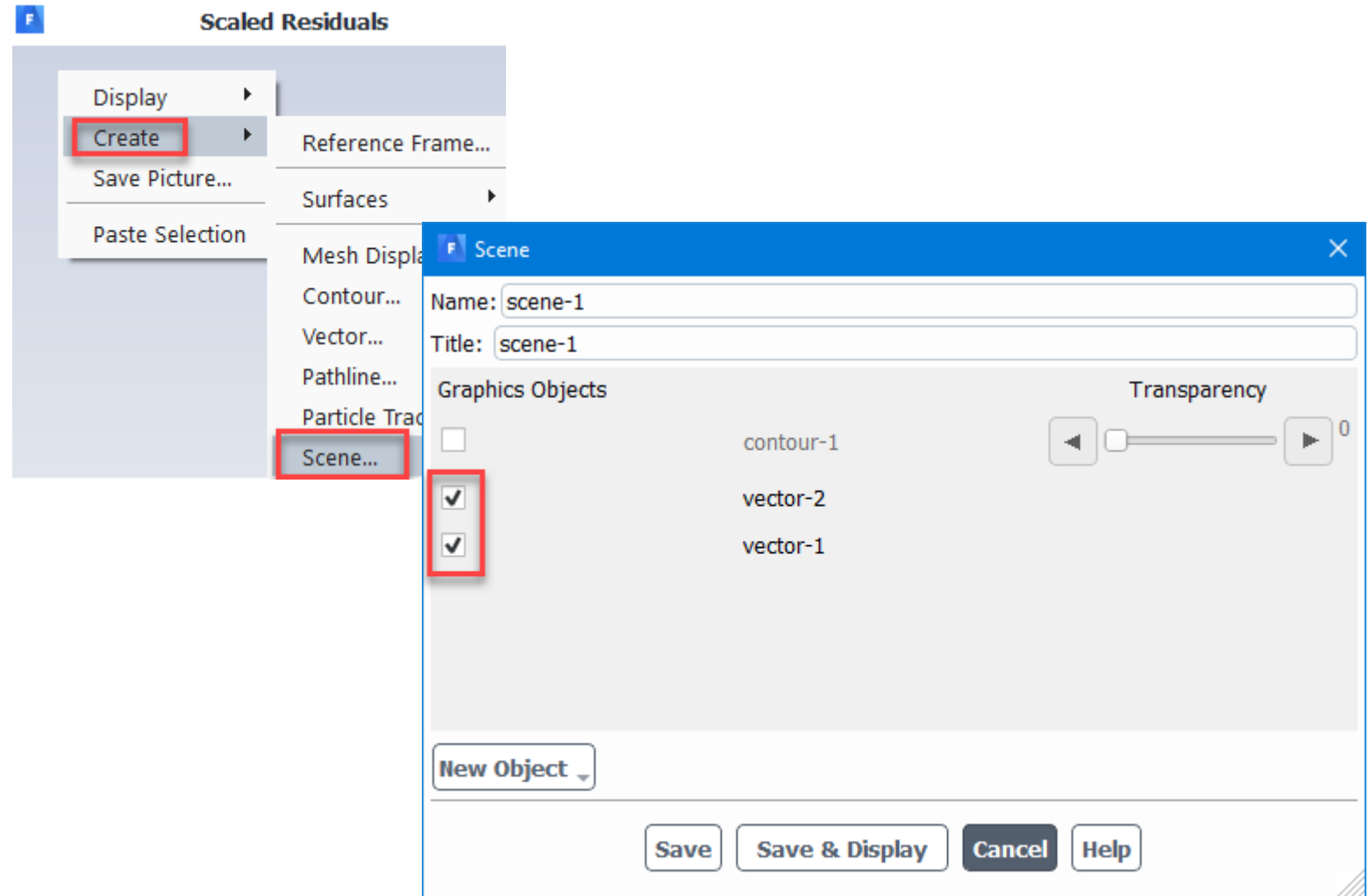
Mid-span Velocity Vector Plot Iso-surface for Zone s1

- Create a *New Vector* plot of *Velocity* on the midspan plane *radial-coordinate-0.22-s1*
- Click *Save/Display*

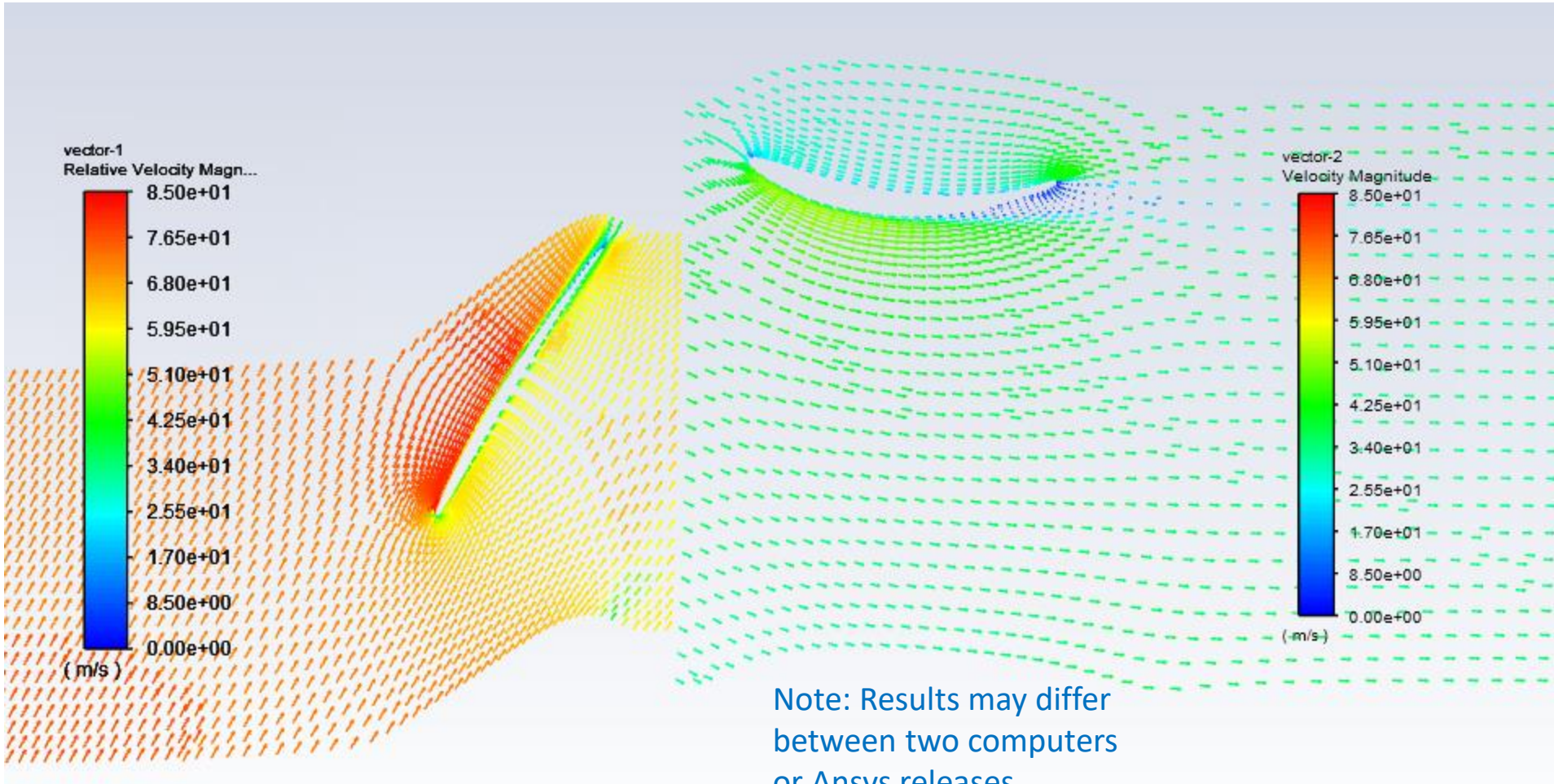


Create a Scene for the 2 Vector Plots

- RMB on an empty in the graphics window and select *Create>Scene...*
- Select both vector plots and click *Save & Display*



Combined Vector Plot for r1 and S1



- When done, write the Fluent .cas and .dat files and exit Fluent

/ Summary

- This workshop has covered:
 - Setting up a steady stage calculation comprising a rotor and a stator
 - Defining a rotating frame
 - Applying rotational periodicity
 - Creating named expressions and report plots for monitoring the pressure rise and the power consumption of the fan rotor
 - Creating a Mixing Plane, General Turbo Interface
 - Solving and monitoring convergence
 - Visualizing the pressure distribution on the impeller walls and the relative velocity vectors at midspan



End of presentation