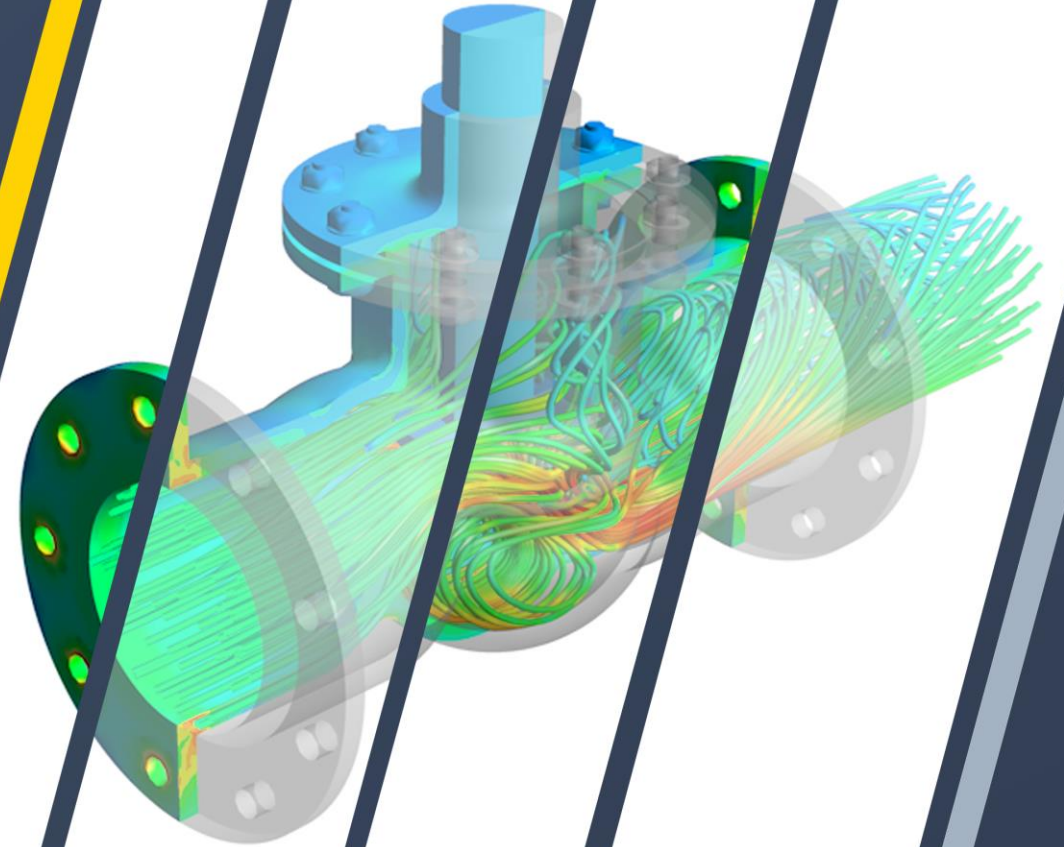




Workshop 04.1: Centrifugal Pump with a Volute

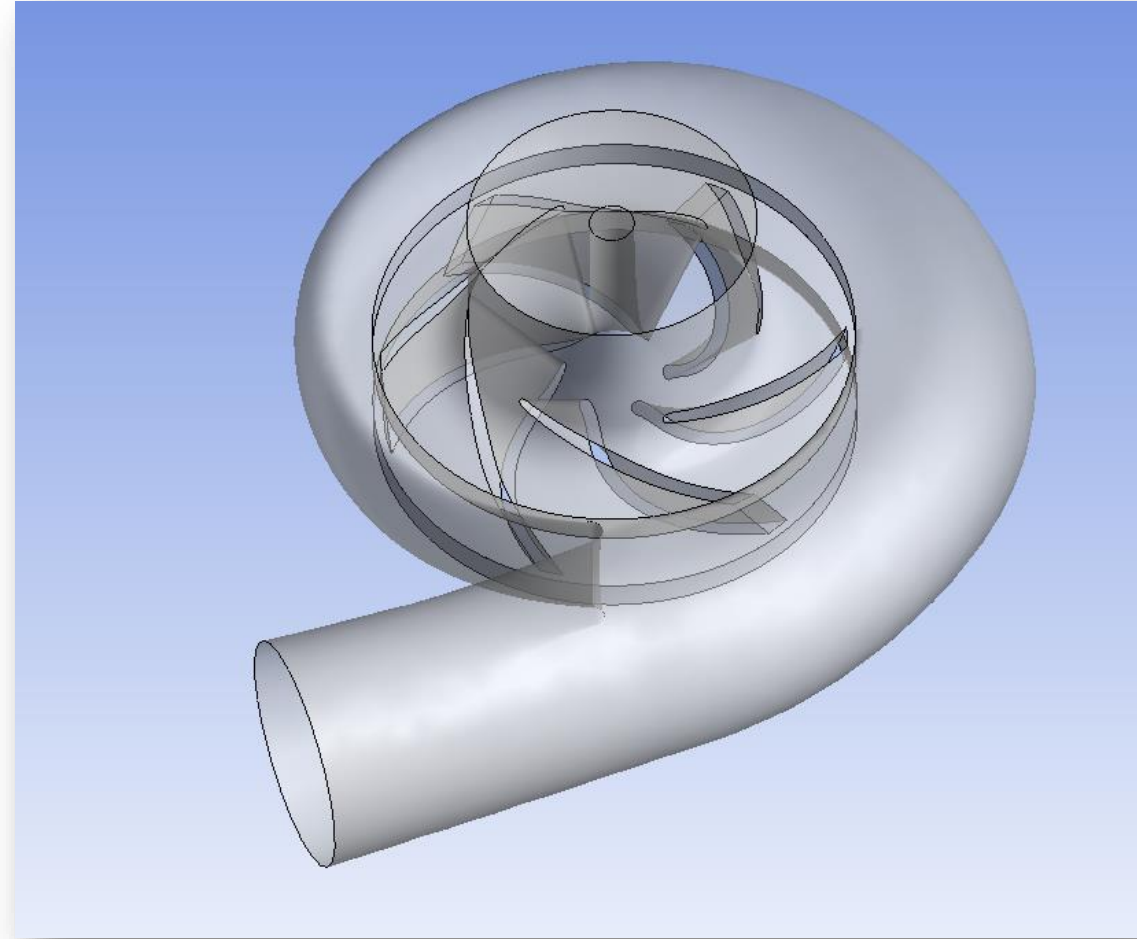
ANSYS CFX Rotating Machinery
Modeling

Release 2019 R3



Introduction

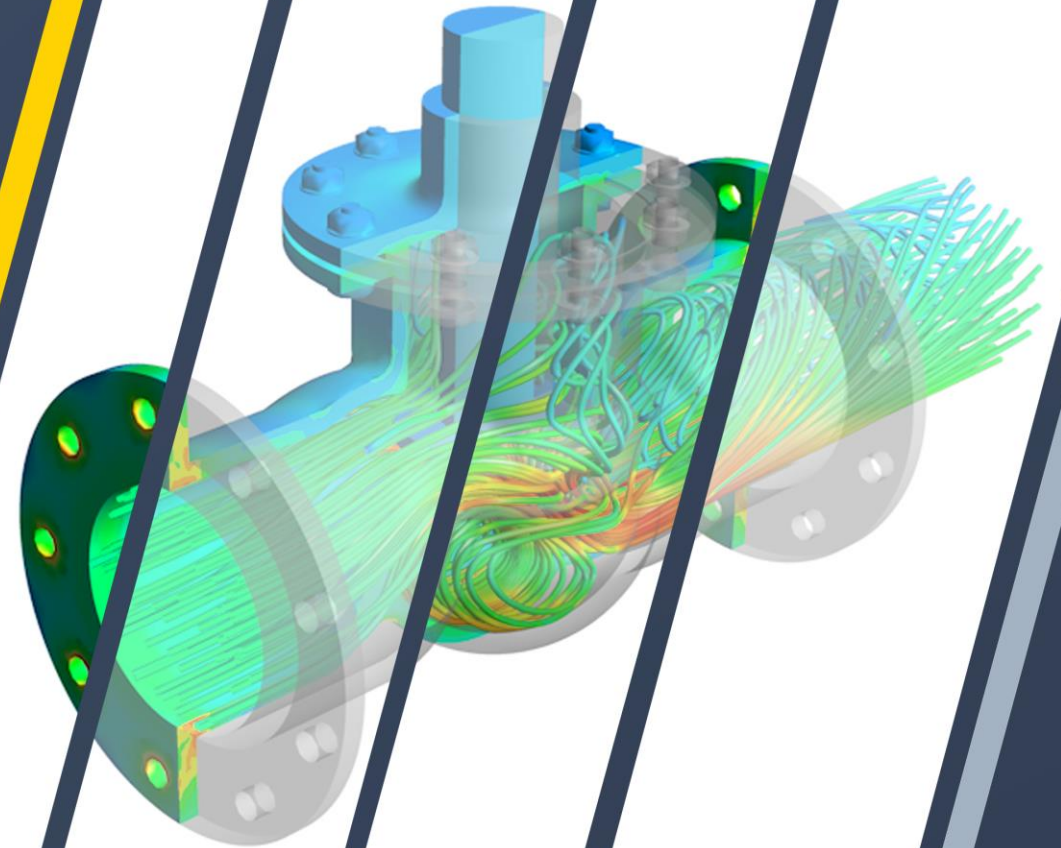
- Workshop Description:
 - This Workshop will illustrate how to setup a simulation involving multiple frames of reference
- Learning Aims:
 - Setting up a case with multiple frames of reference
 - Copying/rotating single passage meshes to create full 360 impeller
 - Setting up domain interfaces
 - Post-processing multiple frames of reference solutions





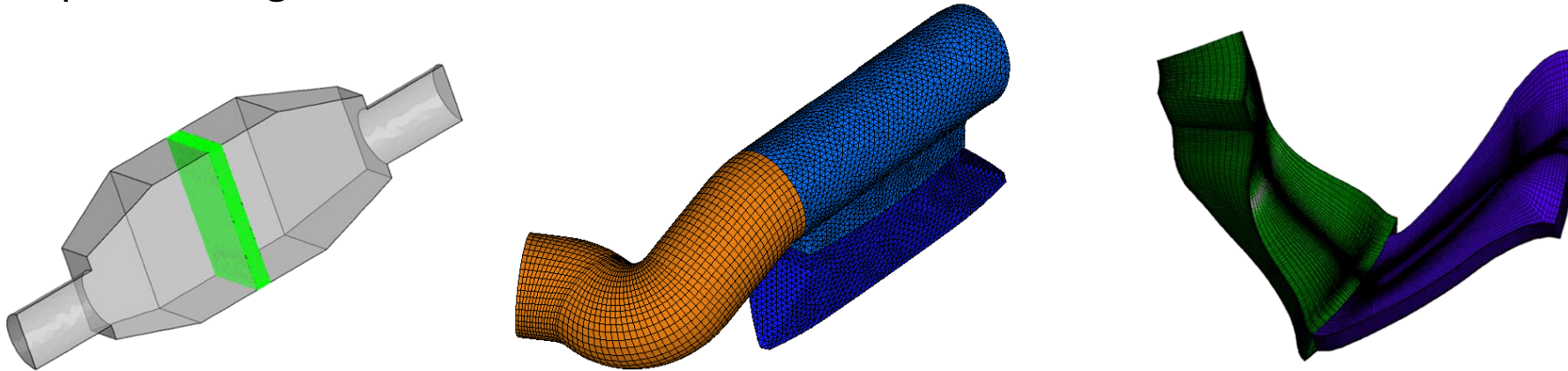
Background

Domain Interfaces



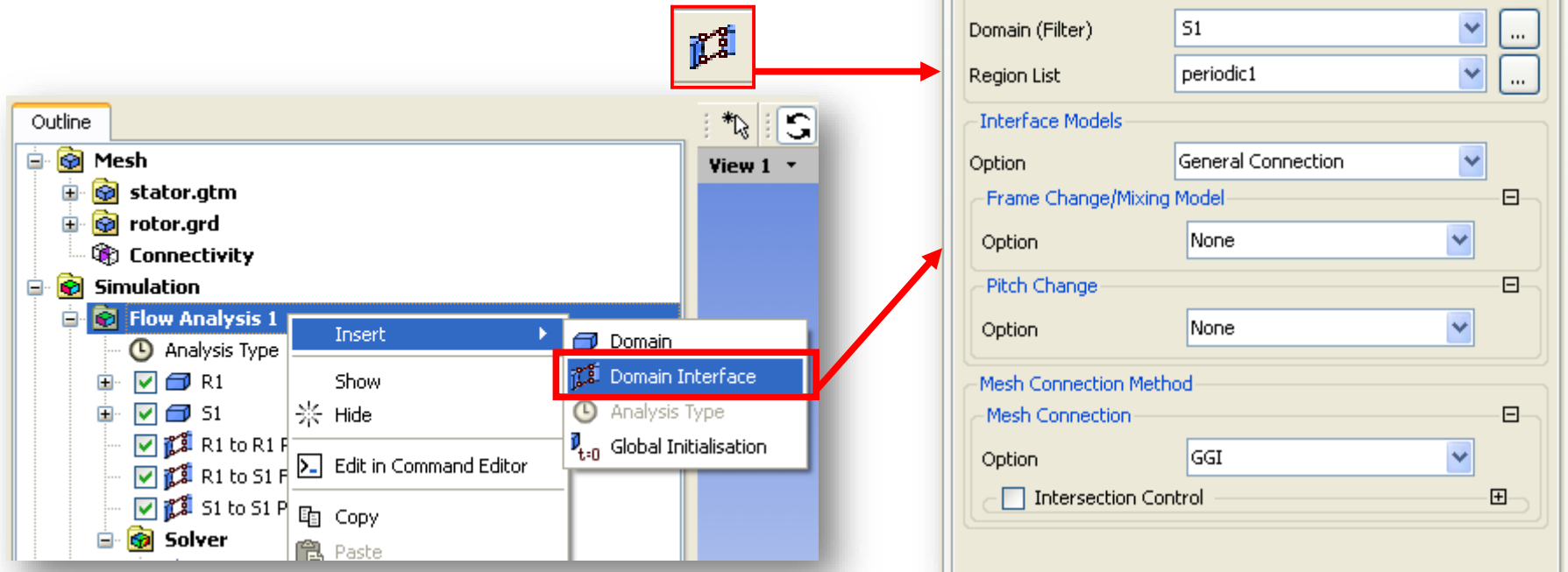
Background

- Recall that domain interfaces are used for:
 - Connection of mismatched meshes (hex to tet for example)
 - a single mesh file may contain non-matching mesh regions and require non-conformal interfaces
 - Changes in reference frames between domains
 - even if the mesh matches
 - Connect different types of domains together (e.g. Fluid to Solid)
 - Create periodic regions within a domain



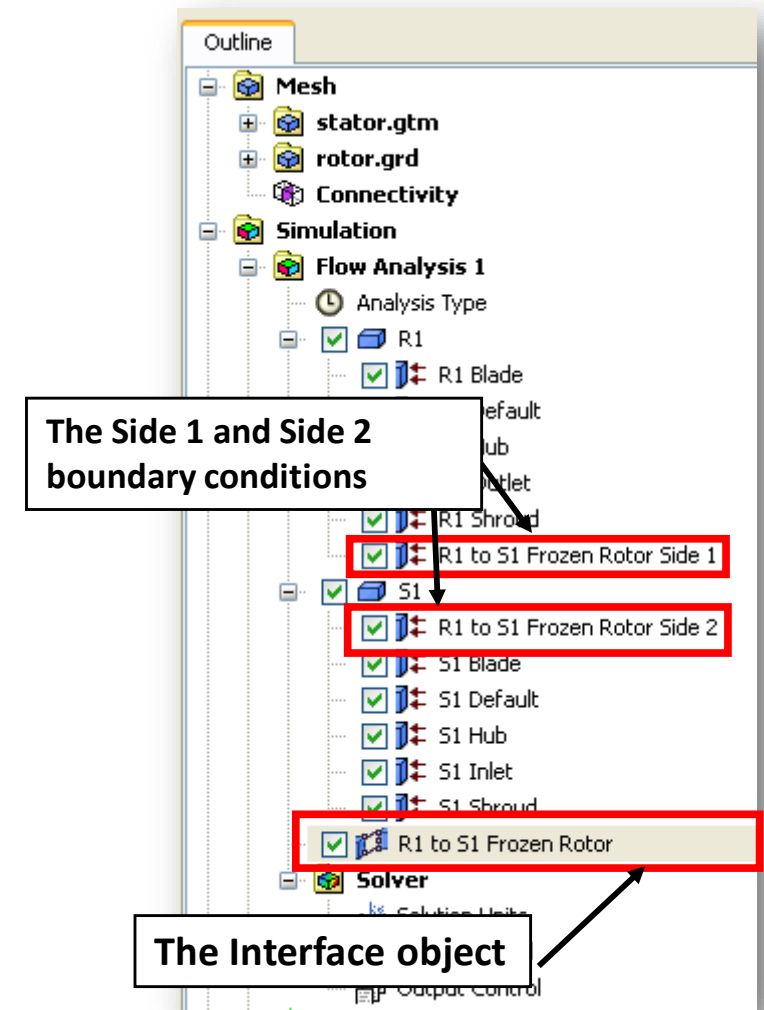
Inserting Domain Interfaces in CFX

- To create a domain interface right-click on the *Flow Analysis* or use the toolbar icon



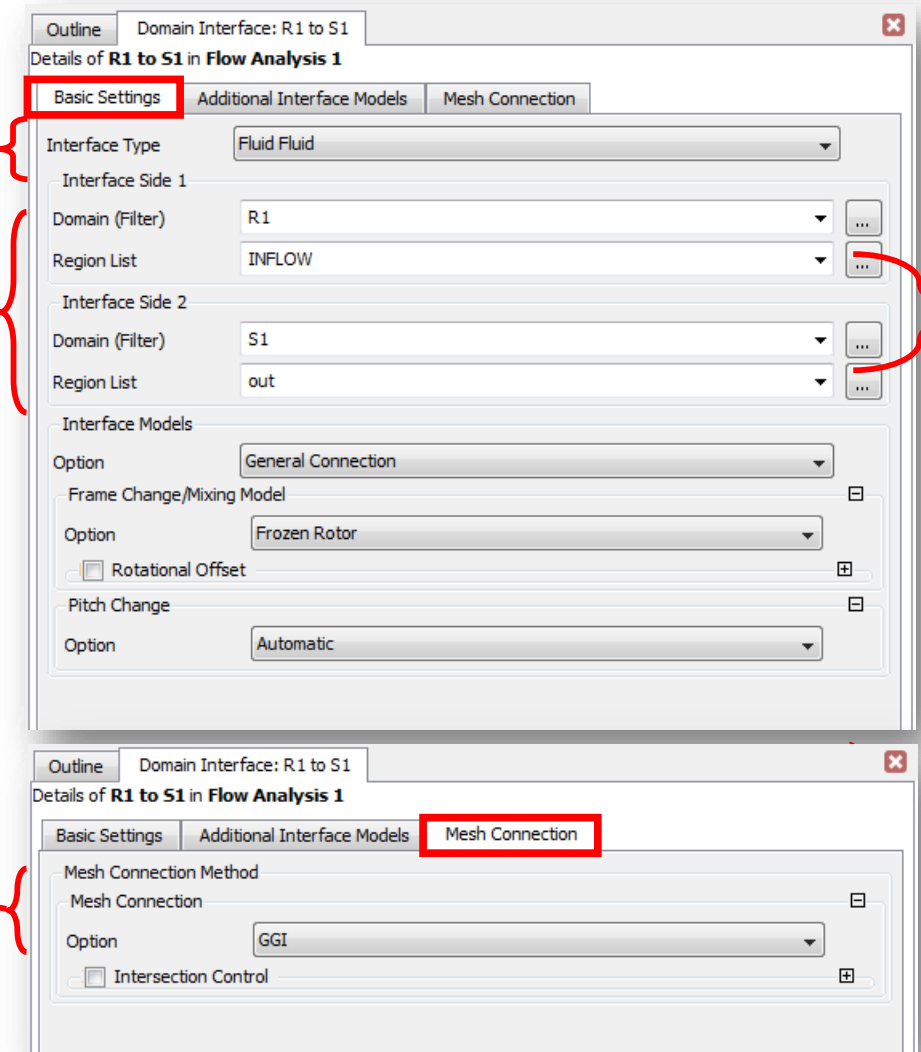
Domain Interfaces and Boundary Objects

- After creating a domain interface 3 new objects are created in the *Outline* Tree
- The interface object is at the *Flow Analysis* level
 - This is the object you should edit to make changes to the domain interface
- Within each domain a *Side 1* or *Side 2* boundary condition is automatically created
 - In general do not edit these objects
 - They will be automatically updated when changes are made to the interface object



Domain Interfaces Panel

- Domain Interfaces connect two sets of surfaces together
 - *Side 1* and *Side 2*
- First select the domain combination to be connected
- Then select the *Side 1* and *Side 2* surface sets
 - The *Domain* (Filter) just limits the scope of the *Region List* to make selection easier
- The *Interface Models* and *Mesh Connection* Method control how data is transferred across the interface

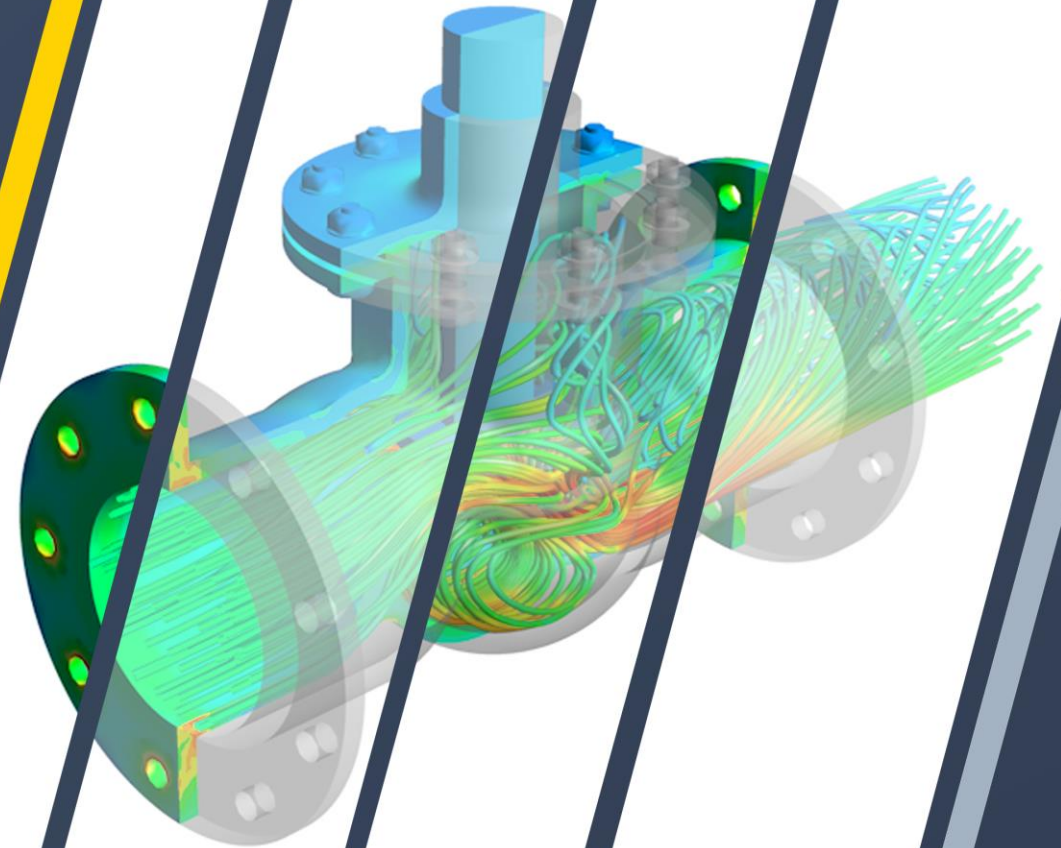


What?

How?

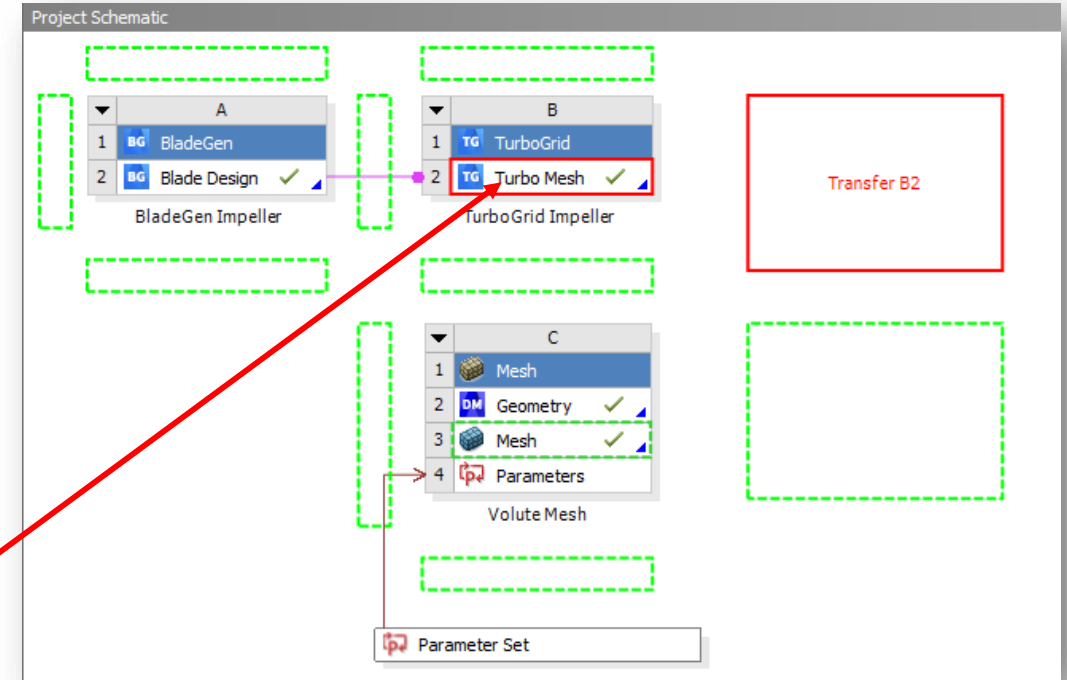


Workshop Instructions



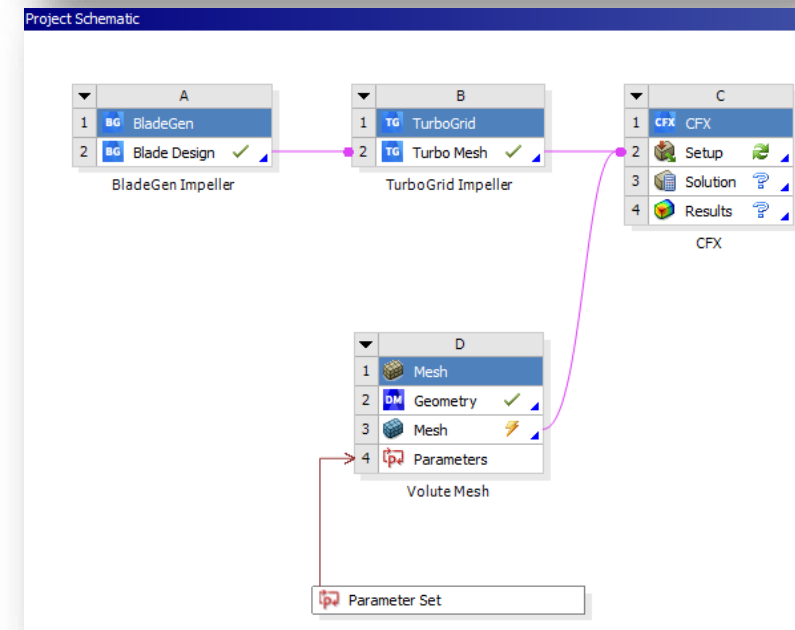
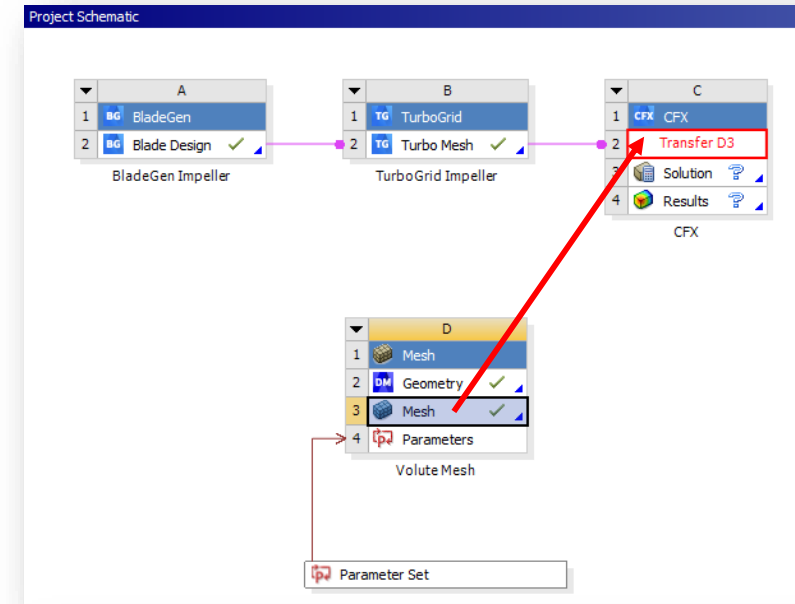
Setup

- Start Workbench and *File>Open* the file: *PumpVolute_mesh.wbpz* provided with the workshop inputs
- In the *Save As* dialogue box edit the *File Name* to *PumpVolute.wbpj* and click *Save*
- This archive contains a mesh for a single passage impeller and a mesh for a volute
- Drag and drop a *CFX Component System* on top of the *Turbo Mesh* cell (B2)



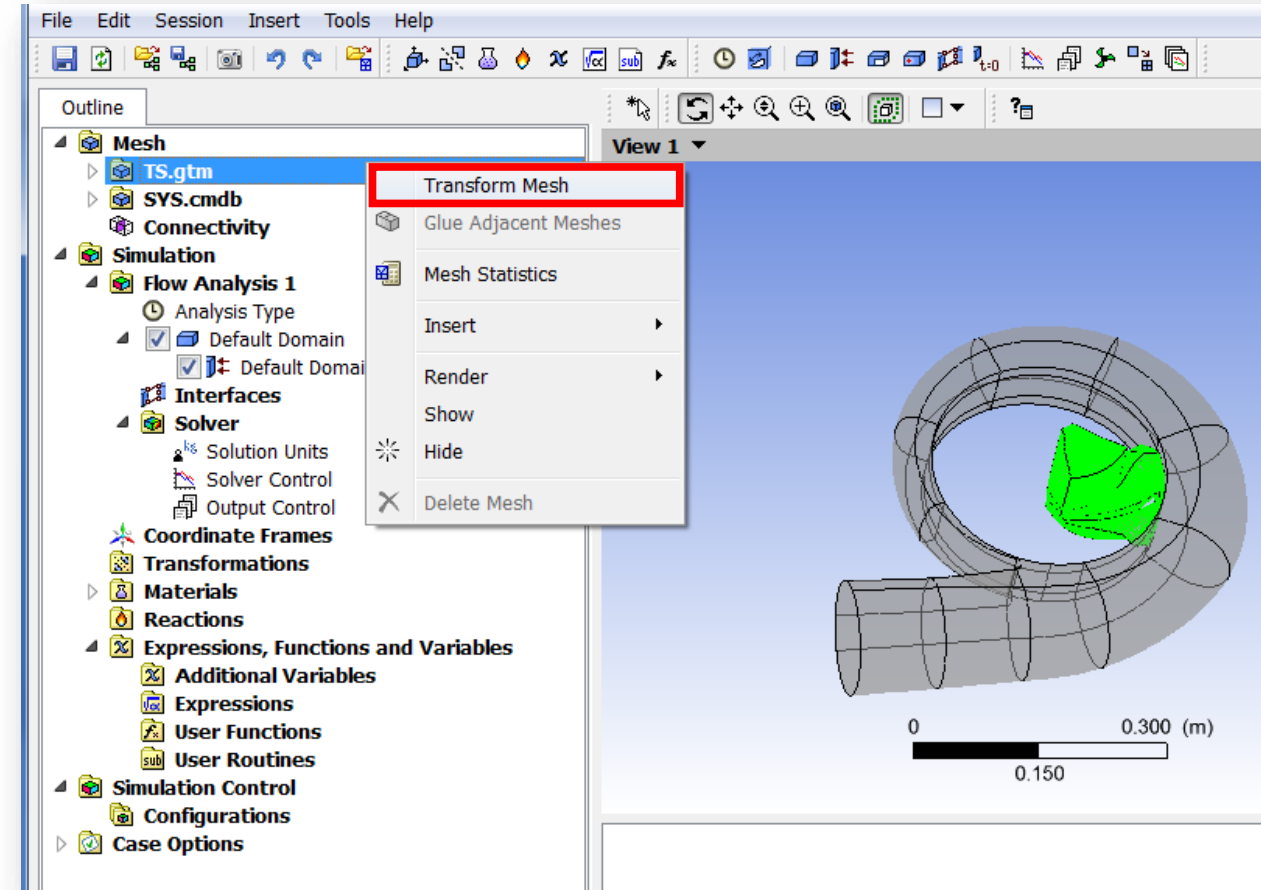
Setup

- Connect the *Volute Mesh (D3)* to the *CFX Setup* cell (C2)
- Right click on the *Volute Mesh* cell (D3) and *Update*
- Double click on the *CFX Setup* cell to open *CFX-Pre*



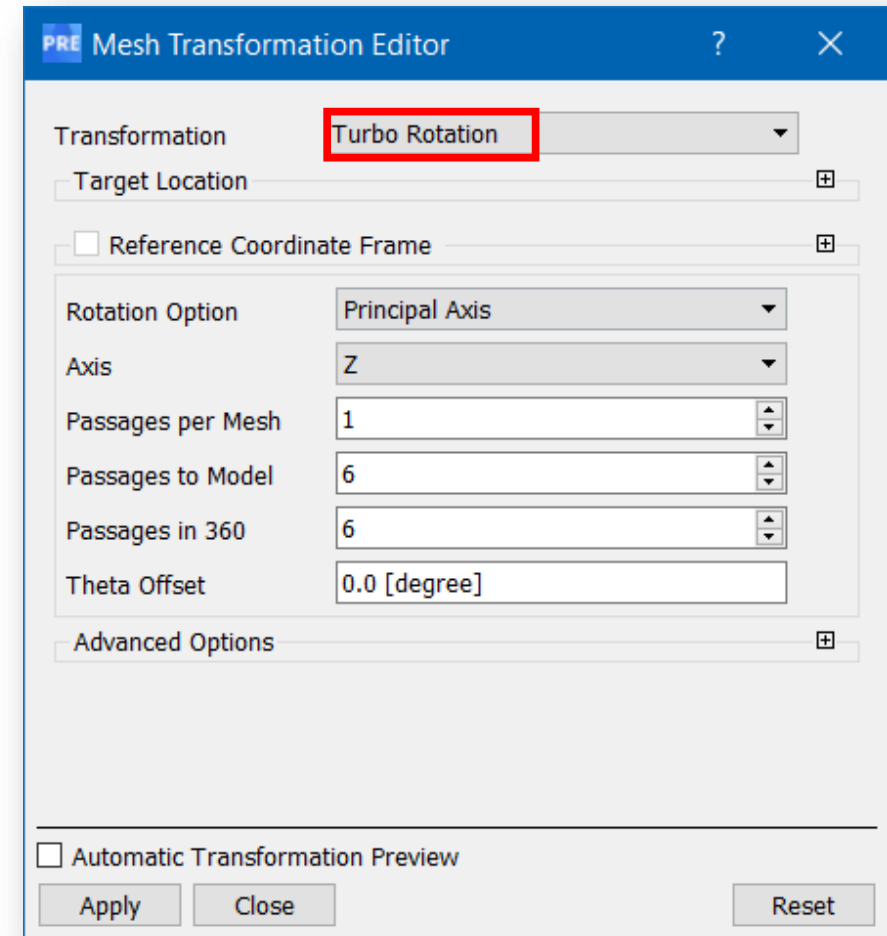
Setup

- When you open *CFX-Pre*, you will see that there is only a single passage of the pump impeller mesh
- We want to have the full 360°
- RMB on *TS.gtm* in *Outline Tree*
 - Select *Transform Mesh*



Setup

- In the *Mesh Transformation Editor*
 - Set *Transformation* to *Turbo Rotation*
 - Set *Passages in 360* to 6
 - Set *Passages to model* to 6
 - Check the *Automatic Transformation Preview* checkbox
 - To verify the mesh to be generated is correct
 - Click *Apply*
 - *Close* the panel



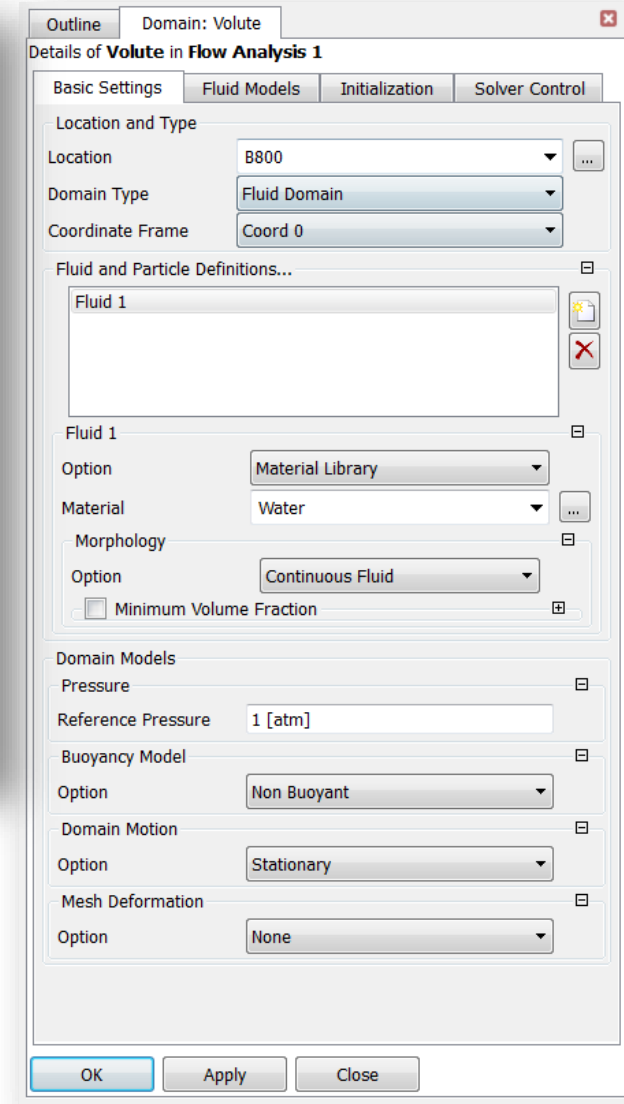
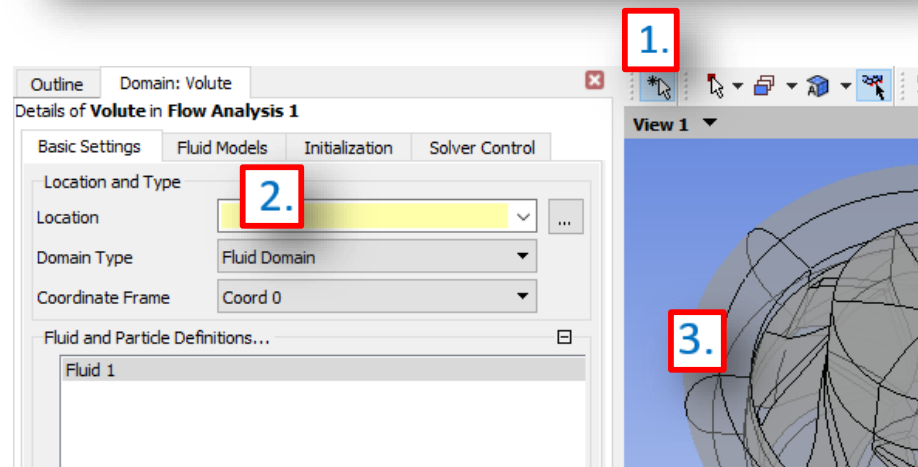
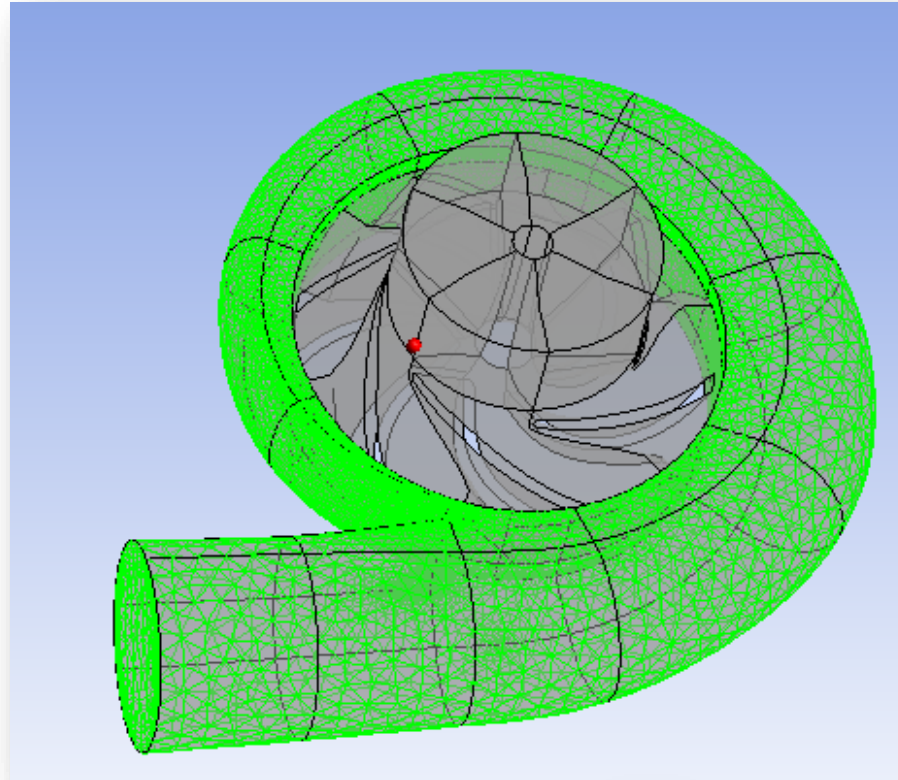
Setup

- Create a domain for the volute:

- *Insert > Domain*
- *Name: Volute*
- *Location: B800 **
- *Material: Water*

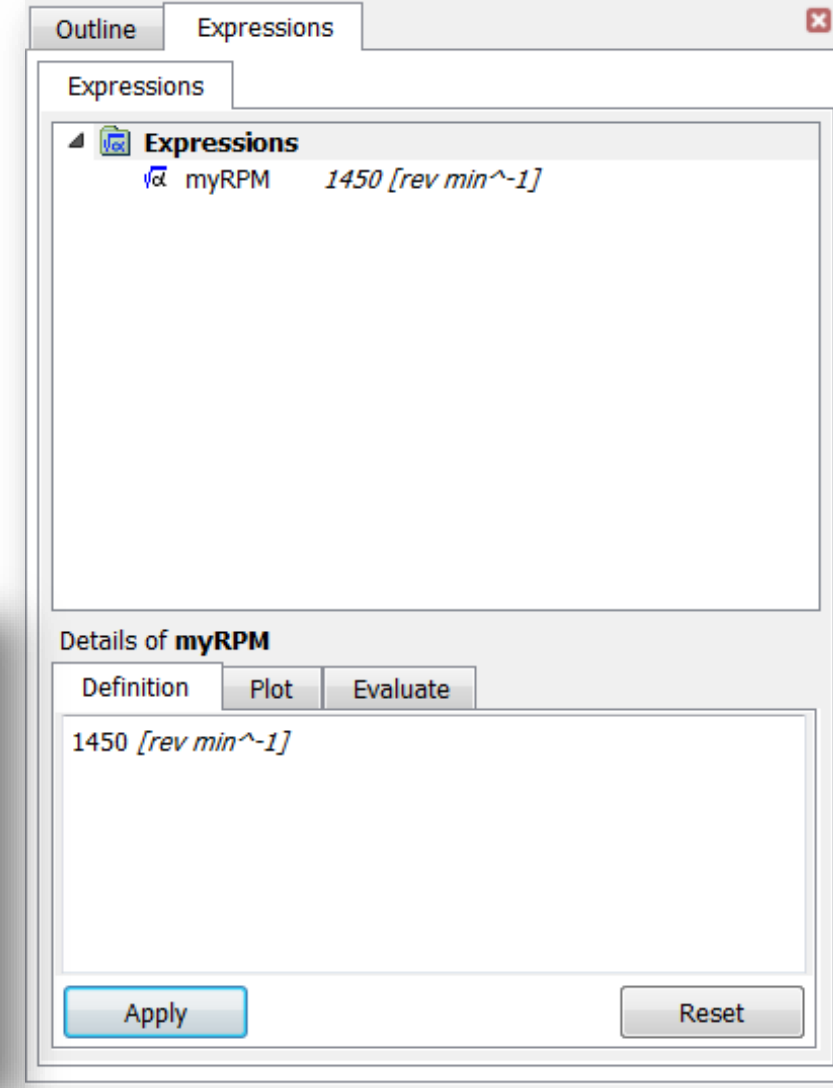
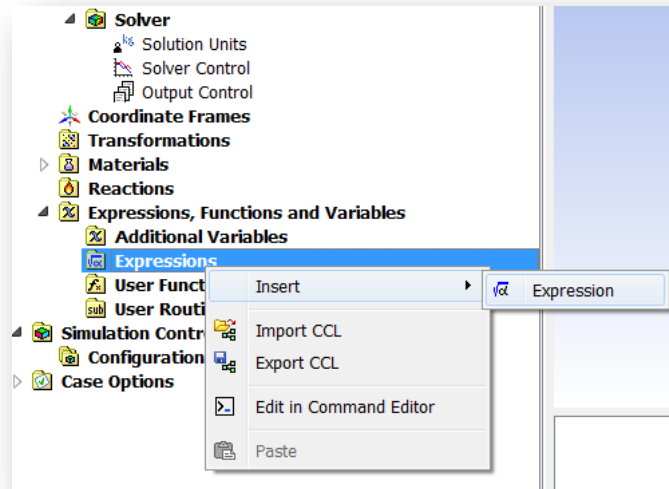
* Alternatively, you may pick the volute mesh location by:

1. *Activating Select*
2. *Clicking in the box next to Location (it will become yellow)*
3. *Clicking on any the volute in the graphics window*



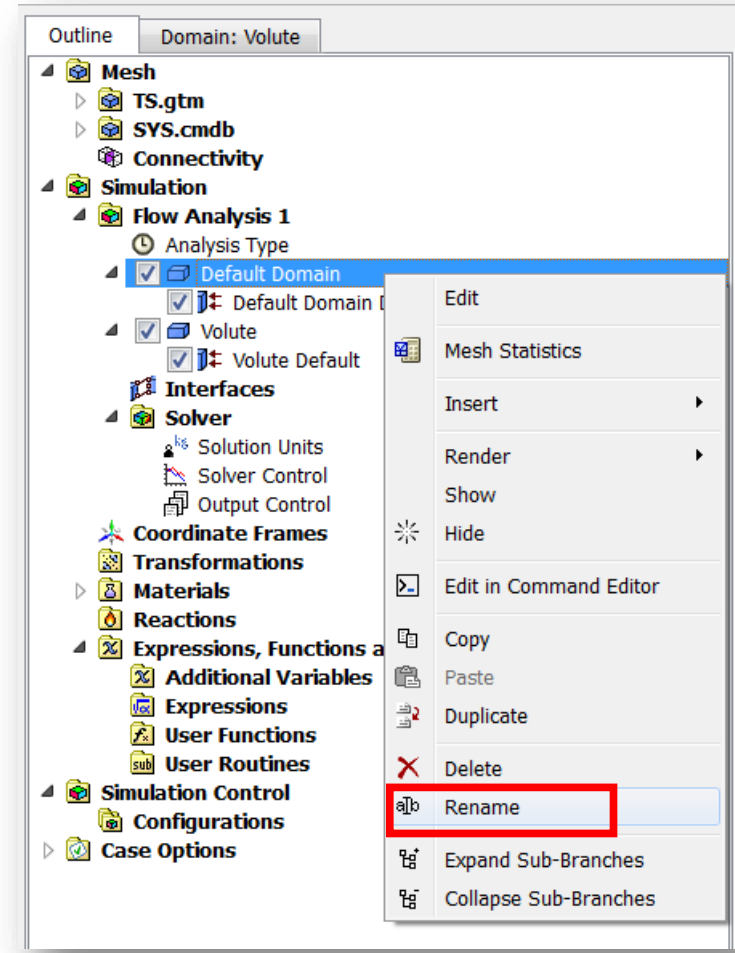
Setup

- Create an expression for rotational speed:
- RMB on *Expressions* in the *Outline*
 - *Insert > Expression*
 - *Name* → *myRPM*
 - Enter *1450 [rev min⁻¹]* and *Apply*




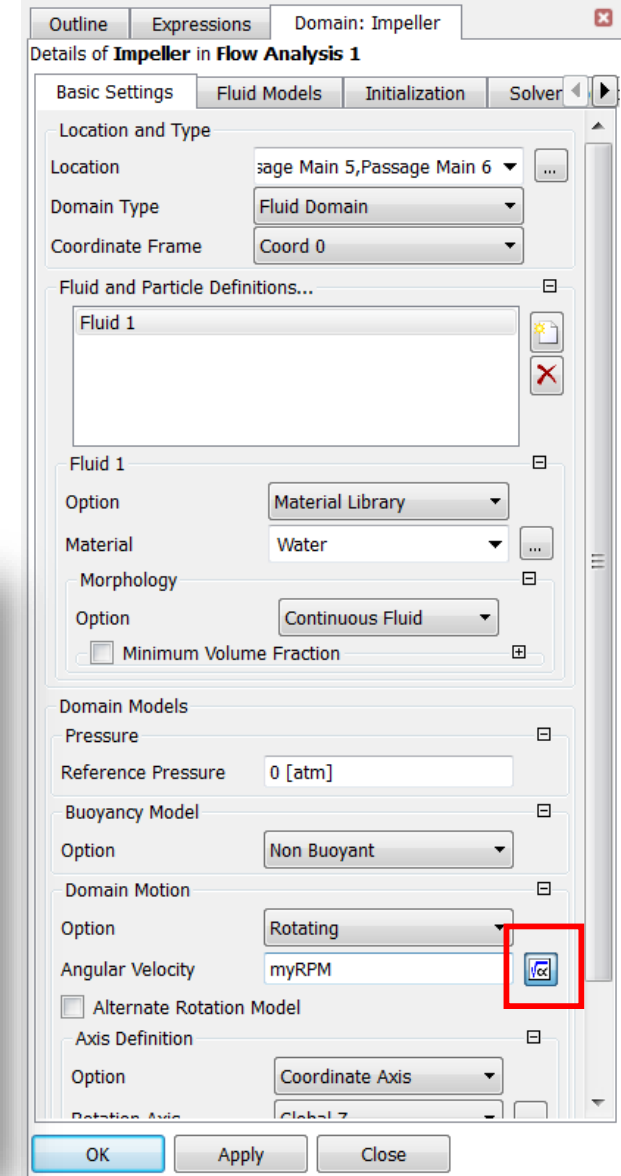
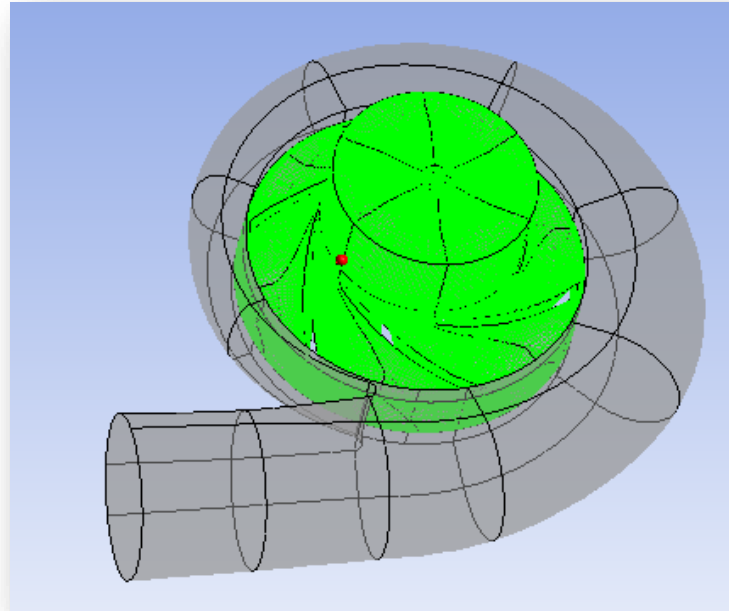
Setup

- Rename *Default Domain*:
 - Right click on *Default Domain*
 - Select *Rename*
 - Type *Impeller* and hit the *Return* key
 - Double click on the *Impeller Domain* in the *Outline* Tree to modify



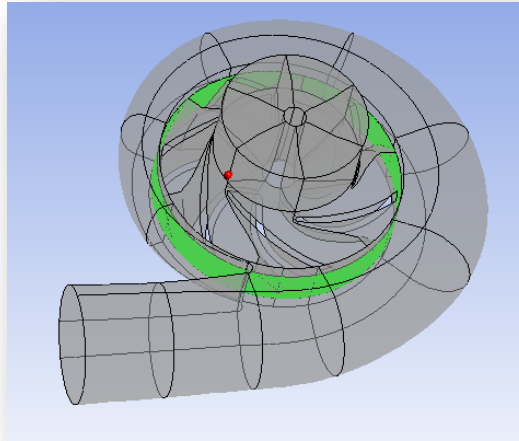
Setup

- *Impeller Domain:*
 - Set *Domain Motion* to *Rotating*
 - Set rotational speed to *myRPM*
 - Use *Enter Expression* button 
 - *Reference Pressure* to 0 [atm]
 - Note that the *Material* is already set to *Water*
 - In the *Fluid Models* tab (not shown)
 - Set *Heat Transfer Option* to *None*
 - Select the *Shear Stress Transport* turbulence model
 - Click *OK* to close panel



Setup

- Create a domain interface between the impeller and volute:
 - RMB on *Interfaces* in *Outline Tree*
 - *Insert > Domain Interface*
 - *Name: Impeller to Volute*
 - *Settings: as shown on the right*



Outline Domain Interface: Impeller to Volute

Details of **Impeller to Volute** in **Flow Analysis 1**

Basic Settings Additional Interface Models Mesh Connection

Interface Type Fluid Fluid

Interface Side 1

Domain (Filter) Impeller

Region List Entire Passage OUTFLOW

Interface Side 2

Domain (Filter) Volute

Region List Inlet 2

Interface Models

Option General Connection

Frame Change/Mixing Model

Option Frozen Rotor

☐ Rotational Offset

Pitch Change

Option Specified Pitch Angles

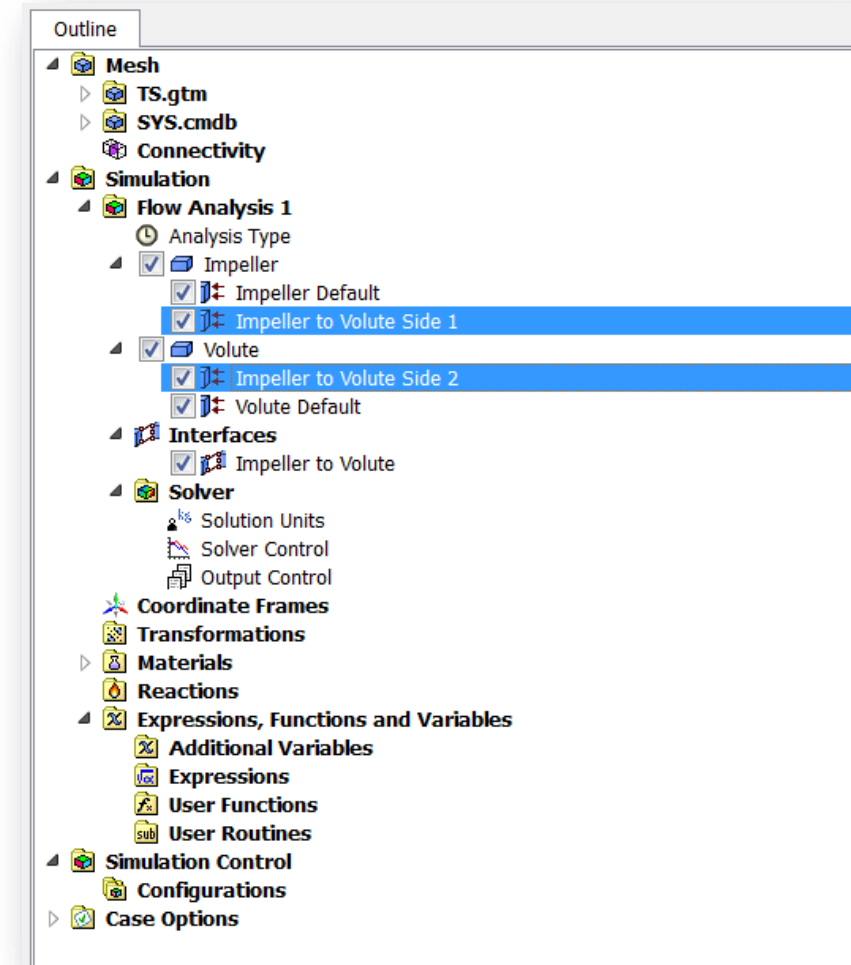
Pitch Angle Side1 360 [degree]

Pitch Angle Side2 360 [degree]

Always specify pitch angles when the interface spans the entire circumference

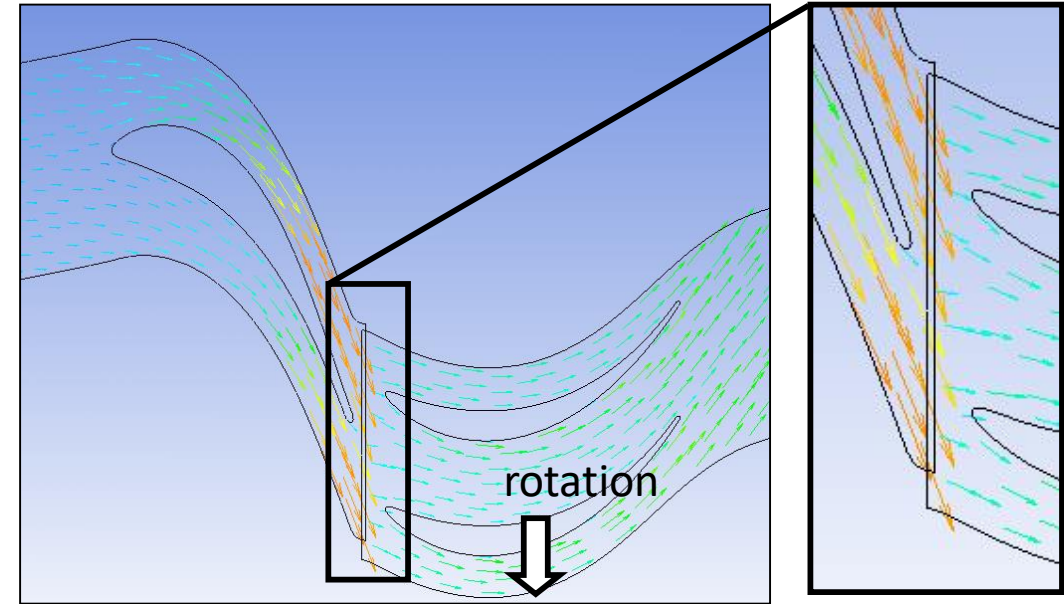
Setup

- Note that *CFX-Pre* has automatically created two separate regions that have the same name as the interface with the suffixes *Side 1* and *Side 2*



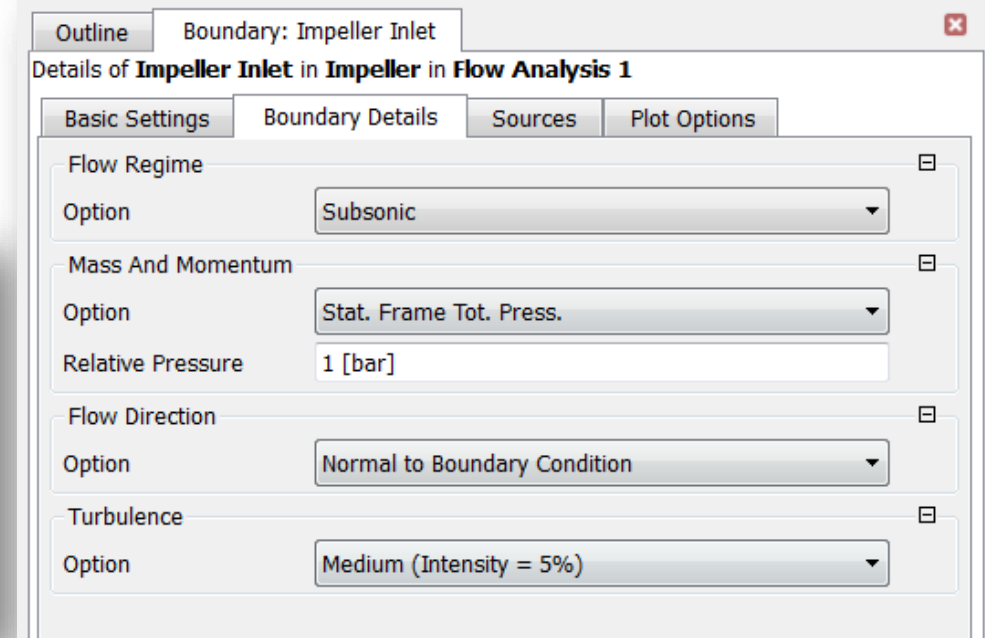
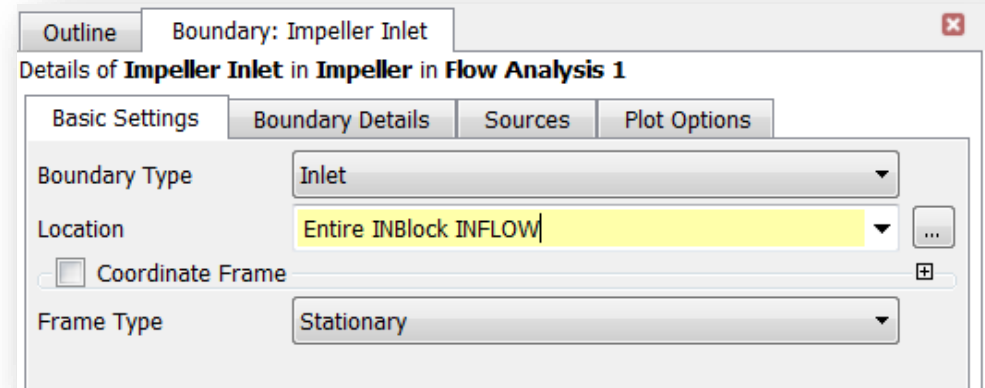
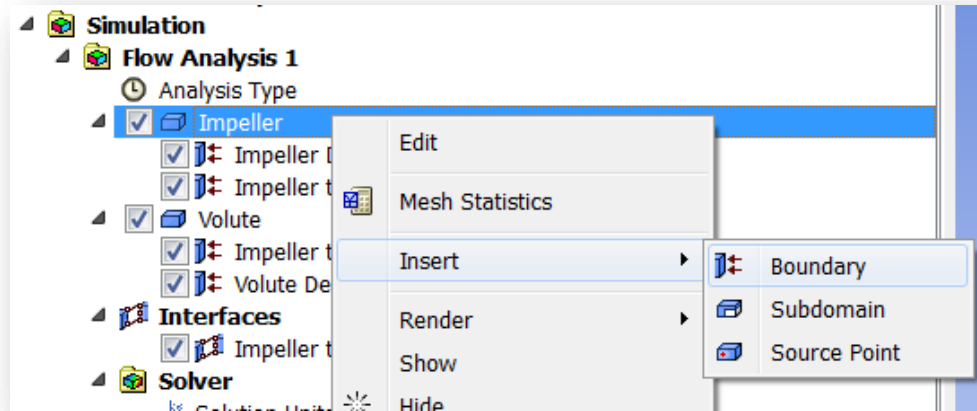
Setup

- Frozen Rotor Concept:
 - Components have a fixed relative position, but the appropriate frame transformation and pitch change is made
 - Most useful when the circumferential variation of the flow is large relative to the component pitch
 - Good approximation for a pump in a volute – each blade passage will observe a different back pressure based on the circumferential position within the volute
- Frozen Rotor Usage:
 - The quasi-steady approximation involved becomes small when the through flow speed is large relative to the machine speed at the interface
 - This model requires the least amount of computational effort of the three frame change models
 - Transient effects at the frame change interface are not modeled



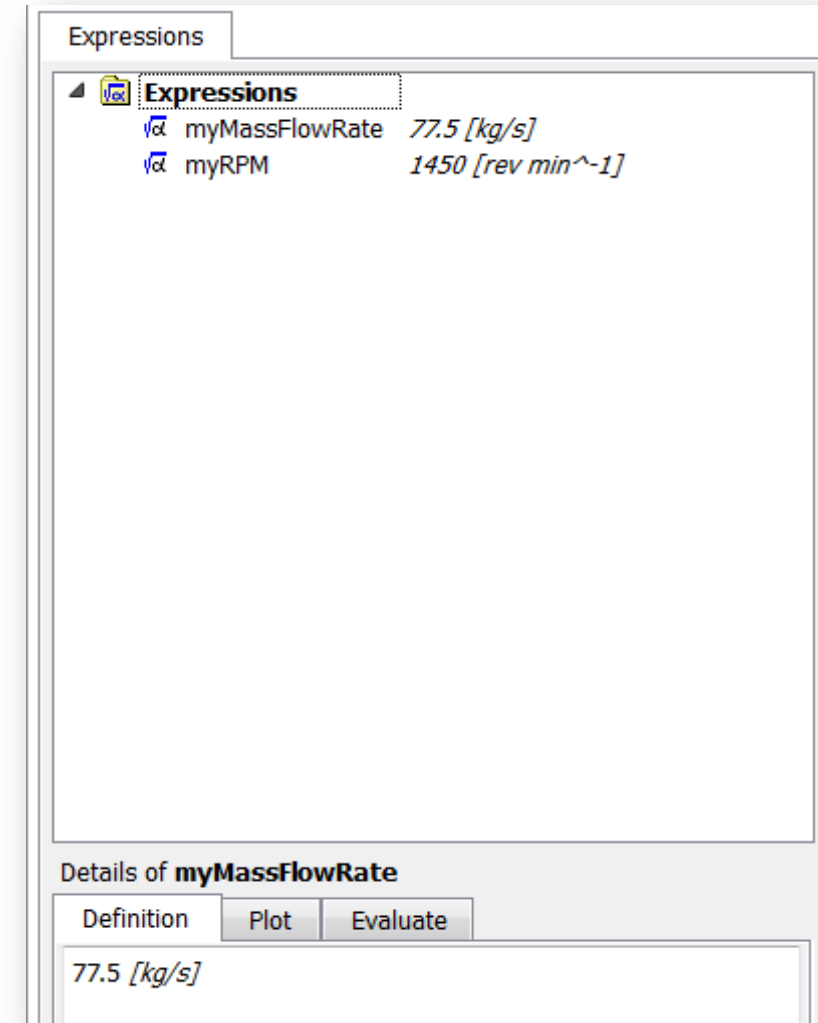
Setup

- Setup Inlet boundary condition:
 - RMB on *Impeller* and *Insert > Boundary*
 - *Name: Impeller Inlet*
 - *Location: Entire INBlock INFLOW*
 - *Mass and Momentum Option: Stat. Frame Tot. Press.*
 - *Relative Pressure = 1 [bar]*
 - Click *OK*



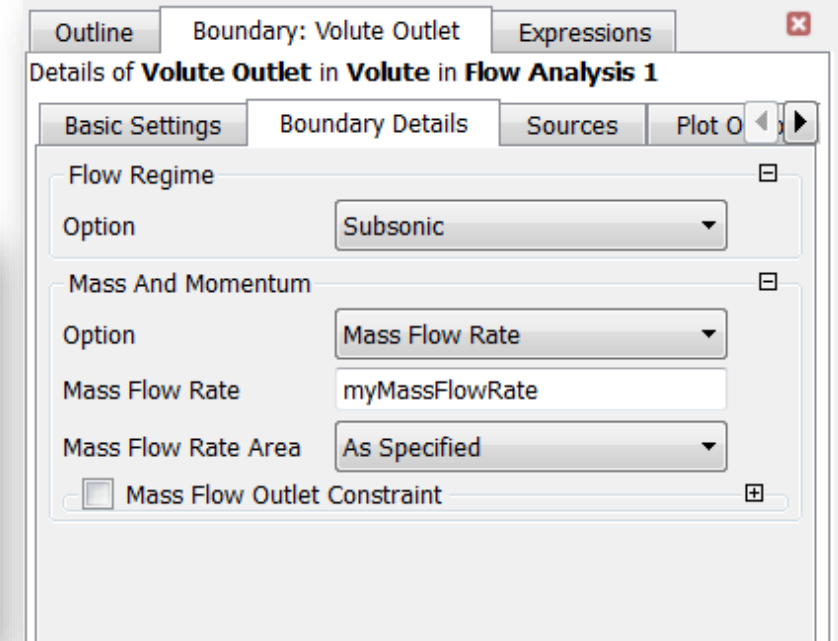
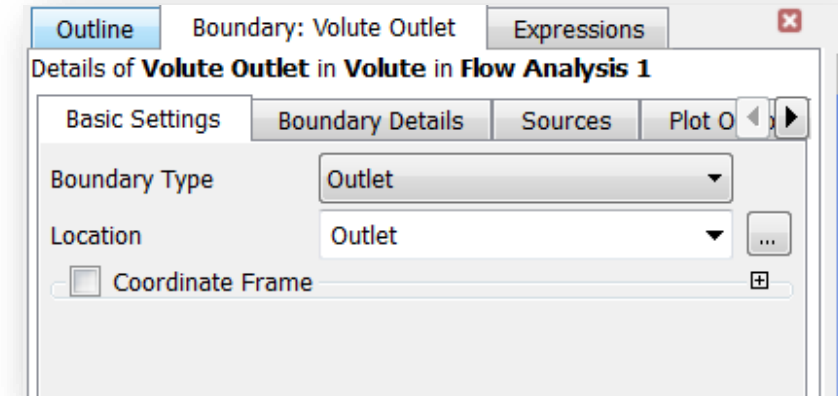
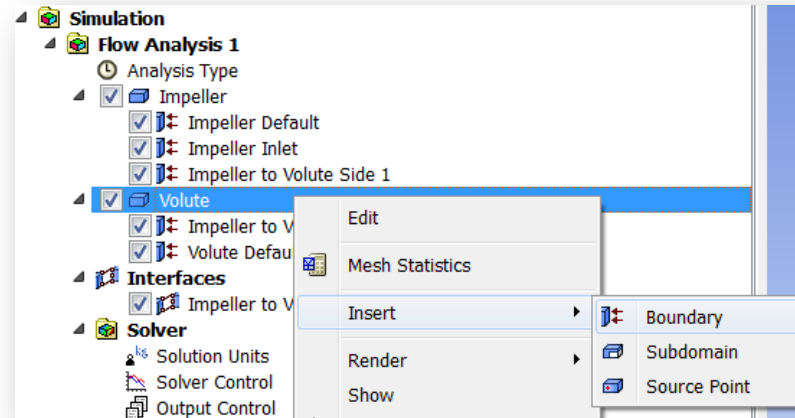
Setup

- Create an expression for mass flow rate
- RMB on *Expressions*
 - *Insert > Expression*
 - *Name: myMassFlowRate*
- Enter *77.5 [kg/s]* and *Apply*



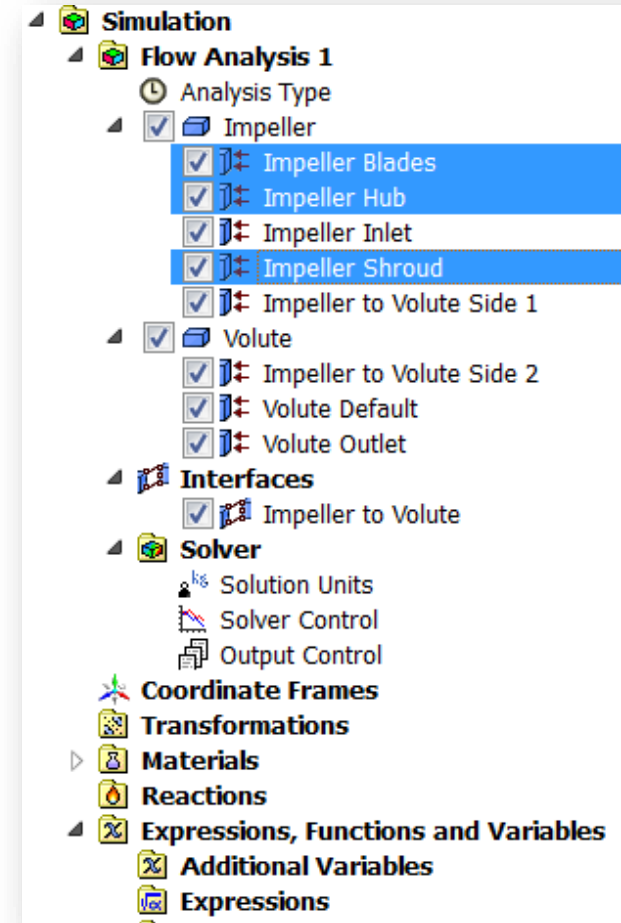
Setup

- Setup Outlet boundary condition:
 - RMB on *Volute* and *Insert > Boundary*
 - *Name: Volute Outlet*
 - *Boundary Type: Outlet*
 - *Location: Outlet*
 - *Mass Flow Rate: myMassFlowRate*



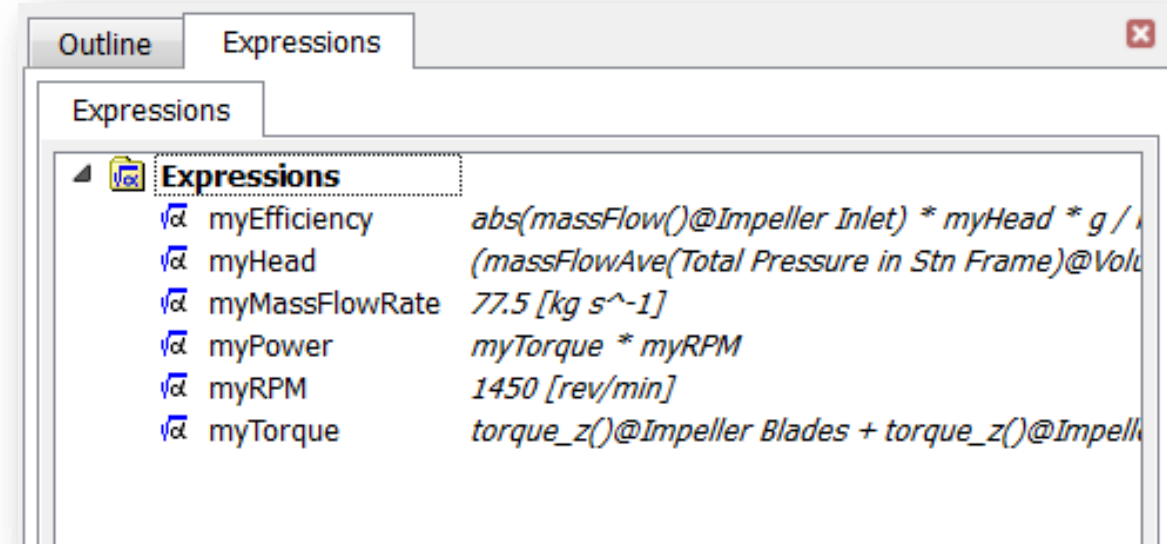
Setup

- Create separate wall regions for:
 - *Impeller Hub*
 - *Regions: Entire INBlock HUB, Entire Passage HUB*
 - *Impeller Shroud*
 - *Regions: Entire INBlock SHROUD, Entire Passage SHROUD*
 - *Impeller Blades*
 - *Region: Entire BLADE*



Setup

- Create Additional Expressions that will be used for monitor points:
 - *myHead*
 - *myTorque*
 - *myPower*
 - *myEfficiency*




$\text{myEfficiency} = \text{abs}(\text{massFlow}()@Impeller Inlet) * \text{myHead} * g / \text{myPower}$

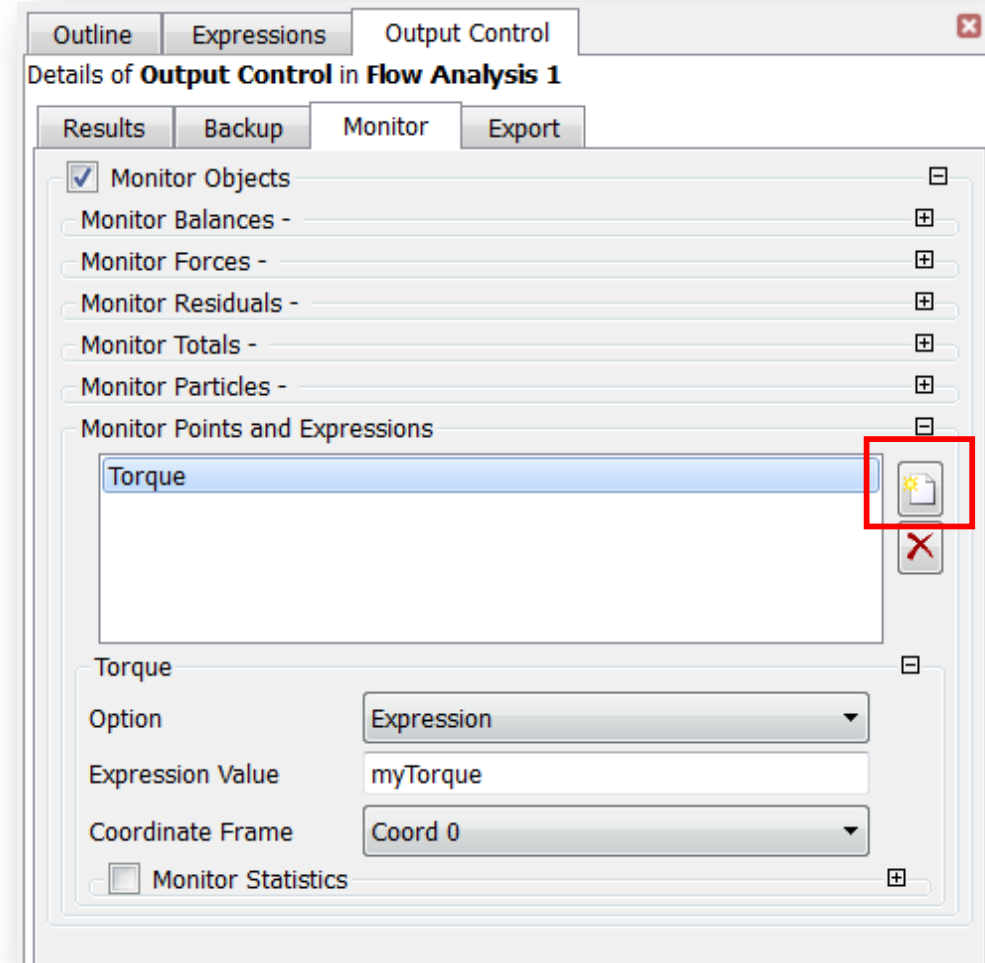
$\text{myHead} = (\text{massFlowAve}(\text{Total Pressure in Stn Frame})@Volute Outlet - \text{massFlowAve}(\text{Total Pressure in Stn Frame})@Impeller Inlet) / (g * \text{ave}(\text{Density})@Impeller Inlet)$

$\text{myPower} = \text{myTorque} * \text{myRPM}$

$\text{myTorque} = \text{torque_z}()@Impeller Blades + \text{torque_z}()@Impeller Hub + \text{torque_z}()@Impeller Shroud$

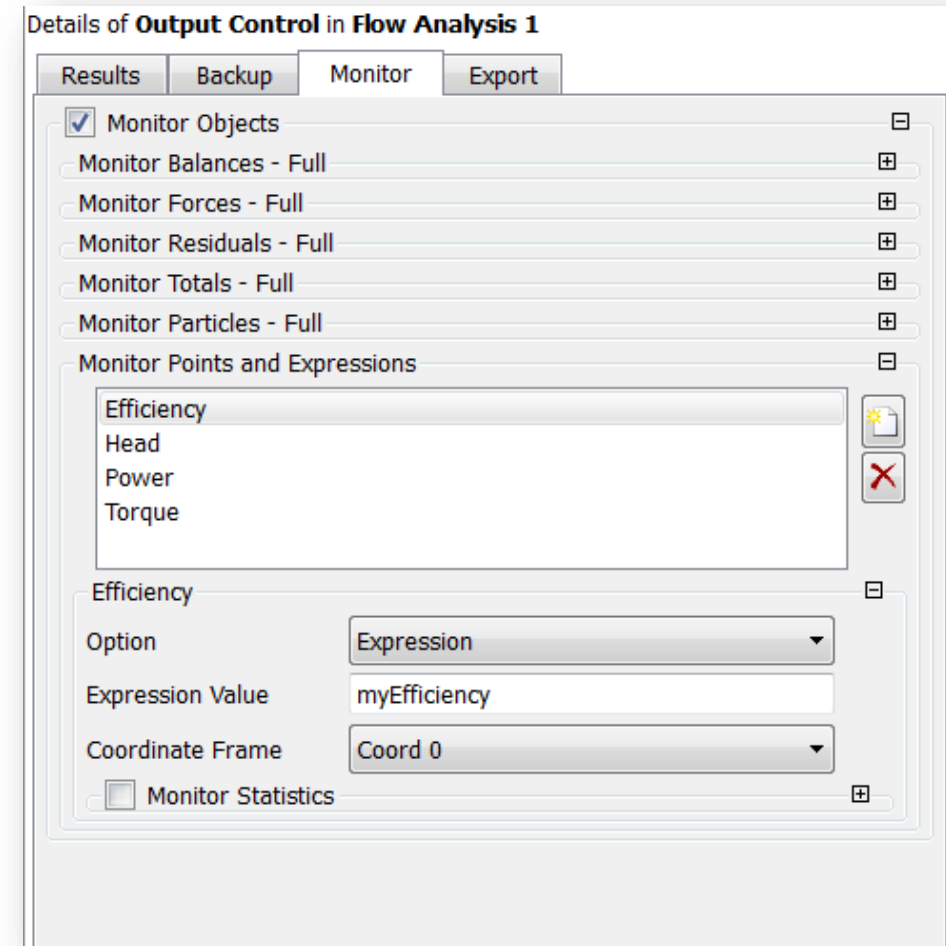
Setup

- Create Monitor Point for Torque
 - Double click *Output Control* in the *Outline Tree*
 - Switch to the *Monitor* Tab
 - Check the *Monitor Objects* checkbox
 - Click the Add New Item icon 
 - Name the Monitor *Torque*
 - Change *Option* to *Expression*
 - Enter *myTorque* as the *Expression Value*
 - Click *Apply*



Setup

- Create monitor points in the same way for *Efficiency, Head and Power*



Setup

- Modify Solver Control
 - A best practice is to set the *Timescale* to be equal to $1/\omega$
 - Change *Timescale Control* to *Physical Timescale*
 - Set *Physical Timescale* to $1[\text{rad}]/\text{myRPM}$
 - Set the *Max. Iterations* to 300
 - Set the *RMS Residual Target* to $1\text{e-}5$
 - Click *Apply*

Details of **Solver Control** in **Flow Analysis 1**

Basic Settings Equation Class Settings Advanced Options

Advection Scheme

Option: High Resolution

Turbulence Numerics

Option: First Order

Convergence Control

Min. Iterations: 1

Max. Iterations: 300

Fluid Timescale Control

Timescale Control: Physical Timescale

Physical Timescale: $1[\text{rad}]/\text{myRPM}$

Convergence Criteria

Residual Type: RMS

Residual Target: $1\text{e-}5$

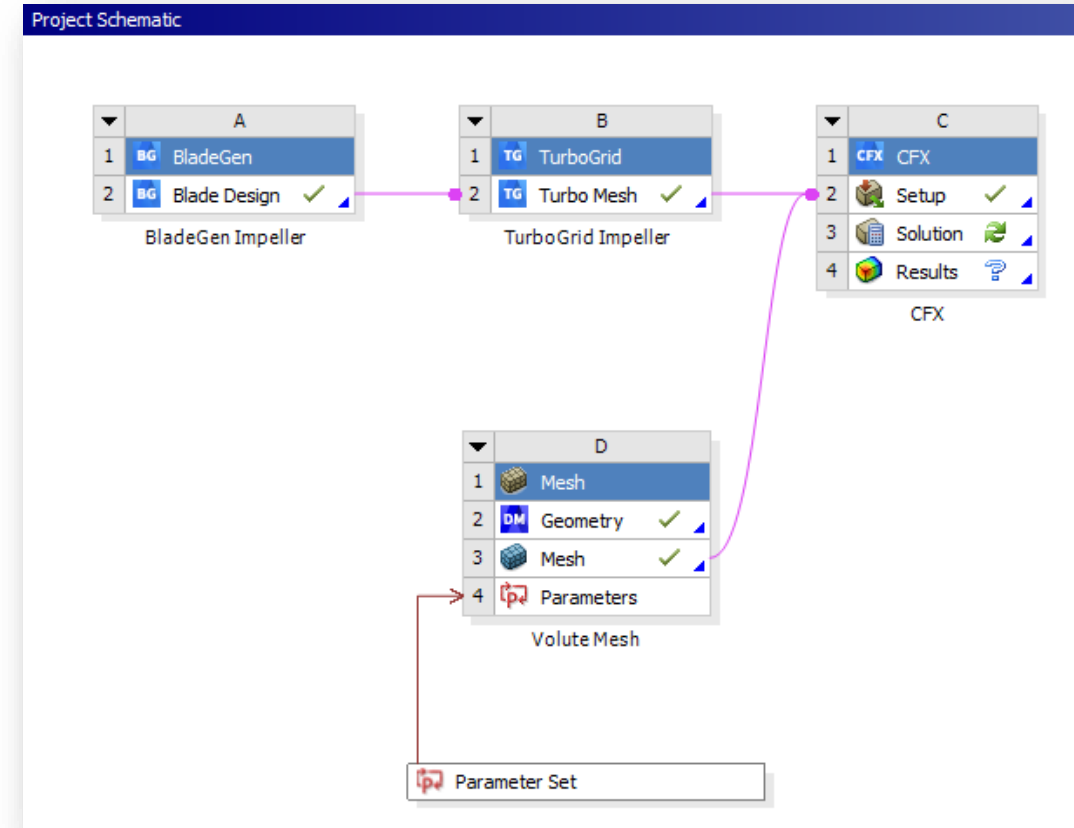
☐ Conservation Target

☐ Elapsed Wall Clock Time Control

☐ Interrupt Control

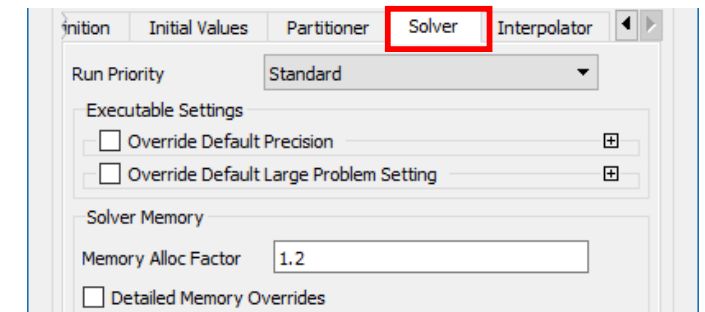
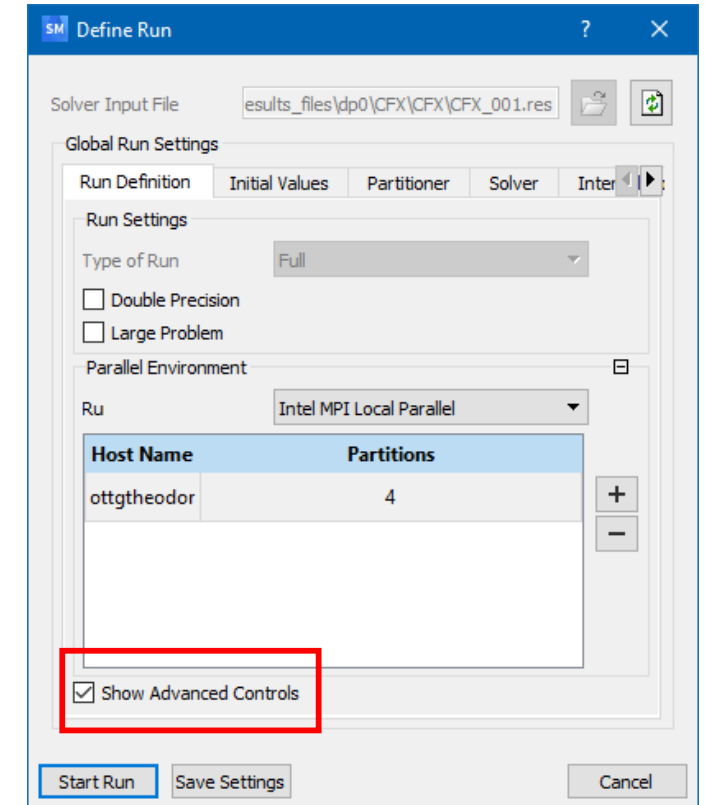
Solver

- Close *CFX-Pre* and go back to the Workbench *Project Schematic*
- Save the *Workbench Project*
- Start the *CFX-Solver*
 - Double Click on the *Solution* cell *C3*



Solver

- On the *Run Definition* Tab, set the *Run Mode* to *Intel MPI Local Parallel* and set the number of *Partitions* to 4
 - Check the *Show Advanced Controls* checkbox
 - In the *Partitioner* and *Solver* tabs set *Memory Alloc Factor* to 1.2 (this is shown below for the *Solver* tab)
 - Click *Start Run*

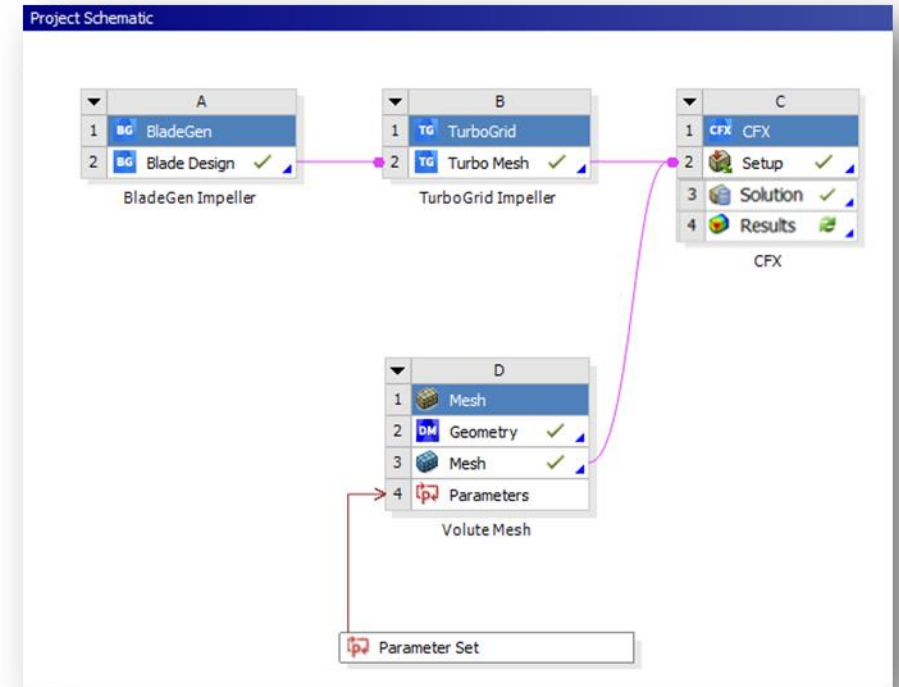


Solver

- Memory Allocation Factor:
 - CFX-Solver is written in Fortran
 - Requires memory to be allocated prior to run
 - Occasionally CFX underestimates the amount of memory required to run the simulation
 - In such cases the CFX-Solver ends up with an error related to not enough memory allocated:
*** INSUFFICIENT MEMORY ALLOCATED ***
 - This could happen in the *Partitioner*, *Interoplator* or *Solver* tasks
 - When this happens, select the corresponding (*Partitioner*, *Interoplator* or *Solver*) Tab in the *Solver Manager* and set the *Memory Alloc Factor* somewhere in the range of 1.2 to 2 as shown in the previous slide for the *Solver*.

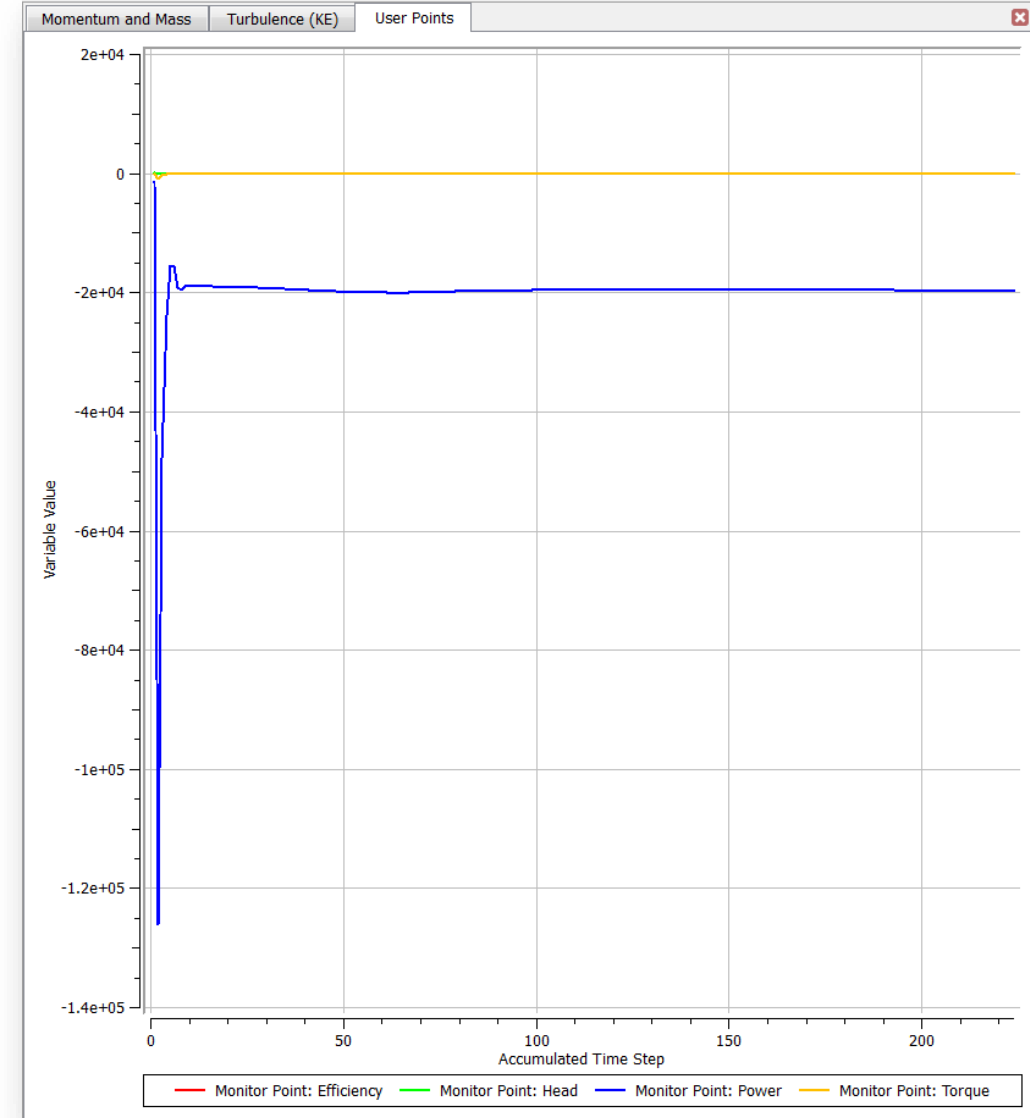
Solver

- This simulation will take roughly 40 minutes to run on 4 cores.
- To save time, you can open the Workbench project named: [PumpVolute_solution.wbpz](#)
 - This project includes a completed simulation



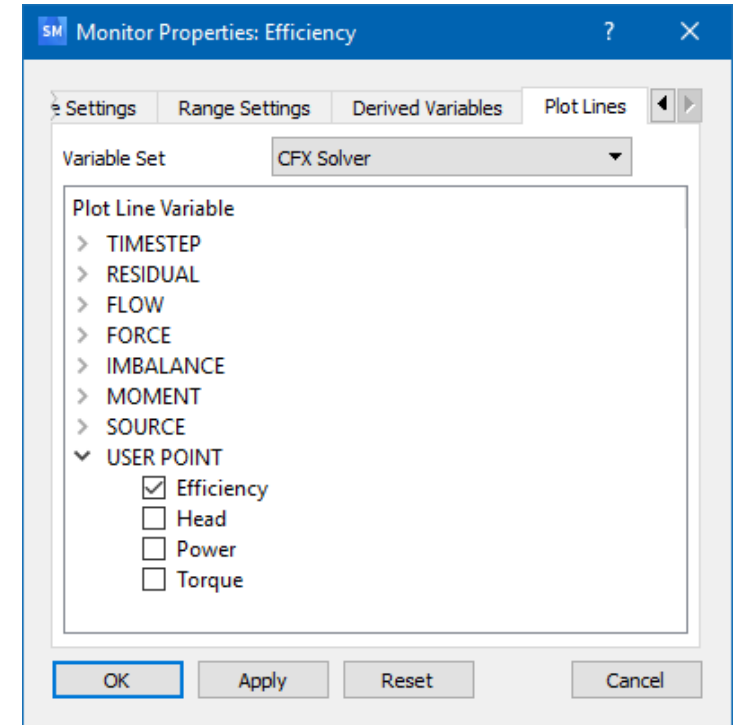
Solver

- If the *Solver Manager* is not already opened, open it by right mouse clicking on the *Solution* cell and selecting *Display Monitors*
 - Switch to the *User Points* Tab
 - We can see monitors for *Efficiency*, *Head*, *Power* and *Torque* that were created
 - Since the scale of these values is so different, let's view efficiency independently



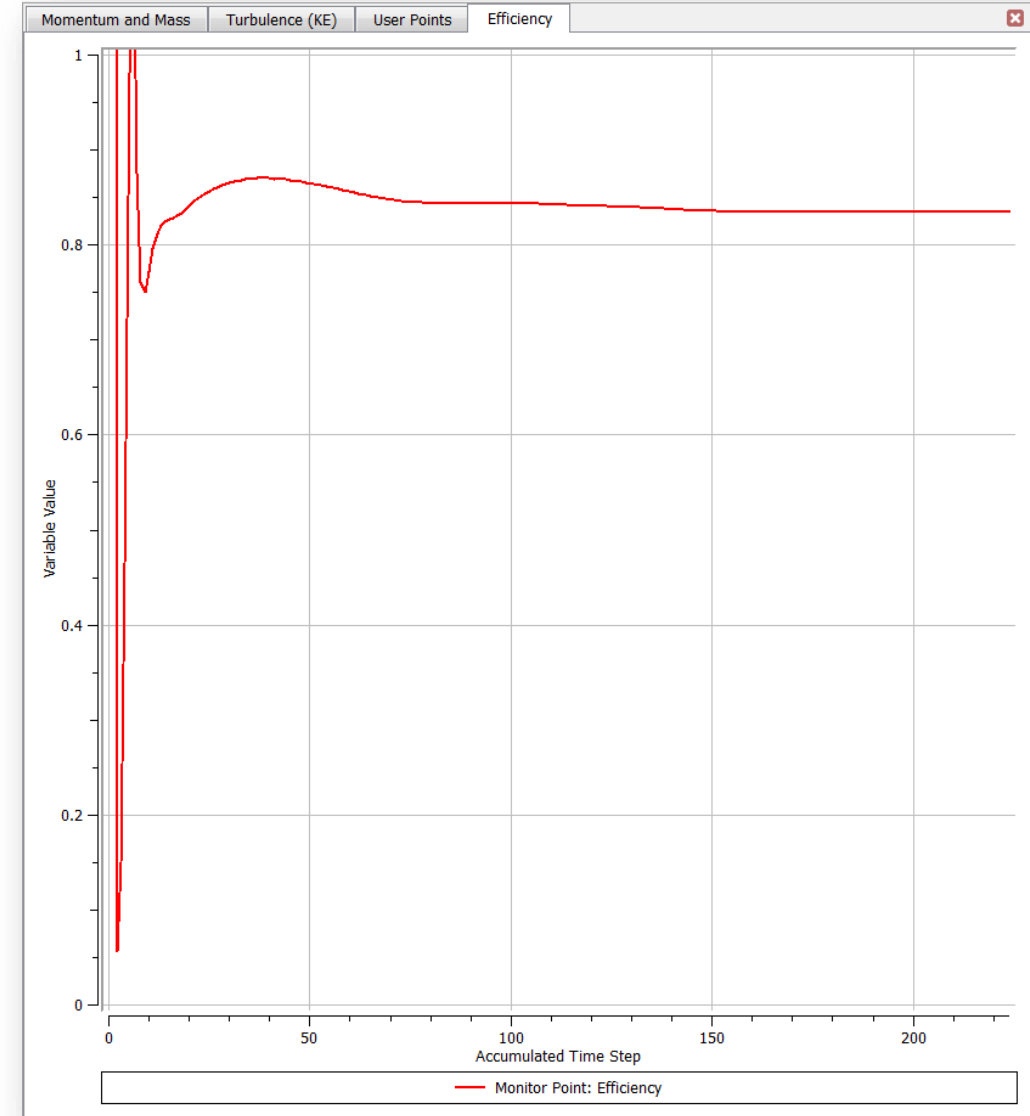
Solver

- *Workspace > New Monitor*
- *Name: Efficiency*
- Under *Plot Lines* Tab, navigate to *User Point Efficiency*
- Under *Range Settings* Tab:
 - *Set Manual Scale (Linear)*
 - *Lower Bound = 0*
 - *Upper Bound = 1*
- Click *OK*



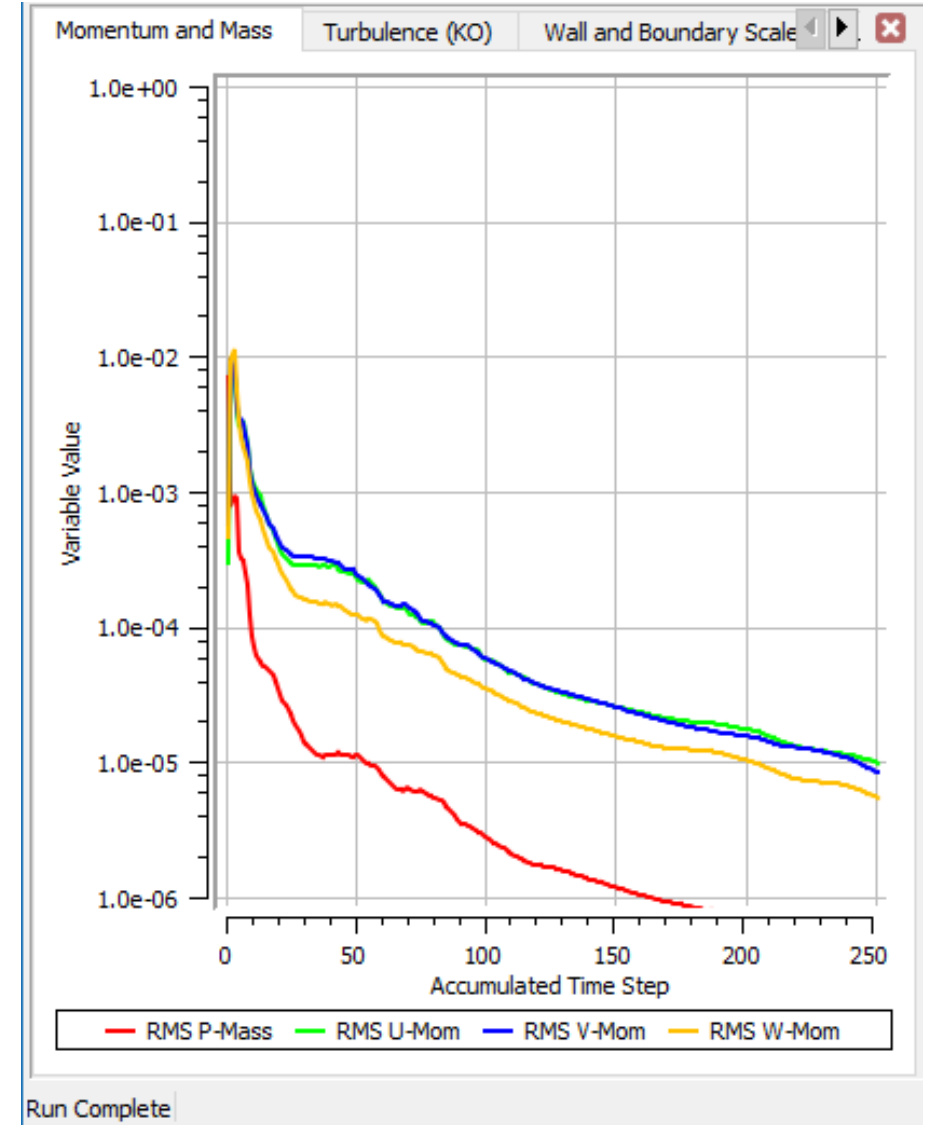
Solver

- We can see that the efficiency has converged to approximately 87%



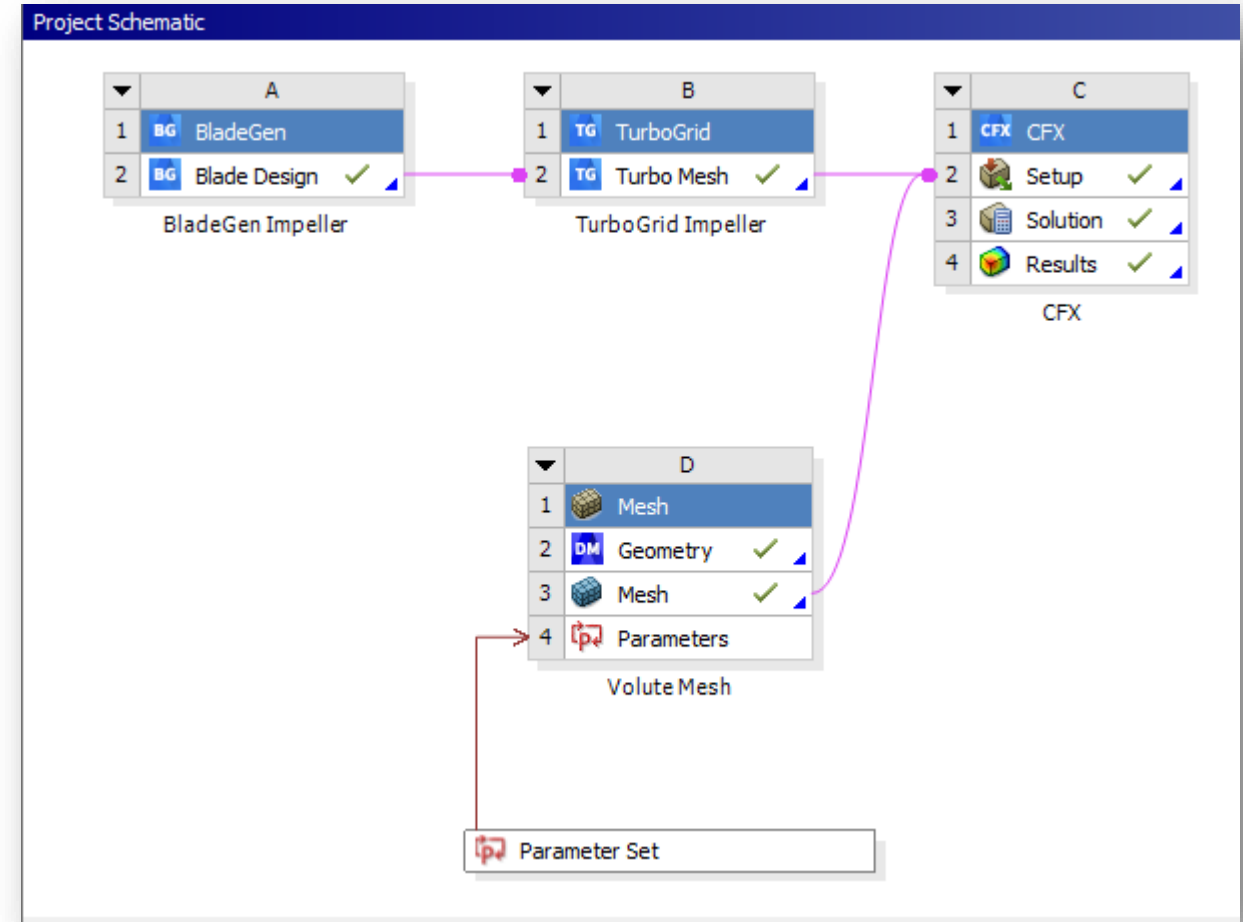
Solver

- Switching to the *Momentum and Mass* Tab, we can see that the residuals are all below $1.0e-5$
- This case is well converged and we can now examine the results in *CFD-Post*



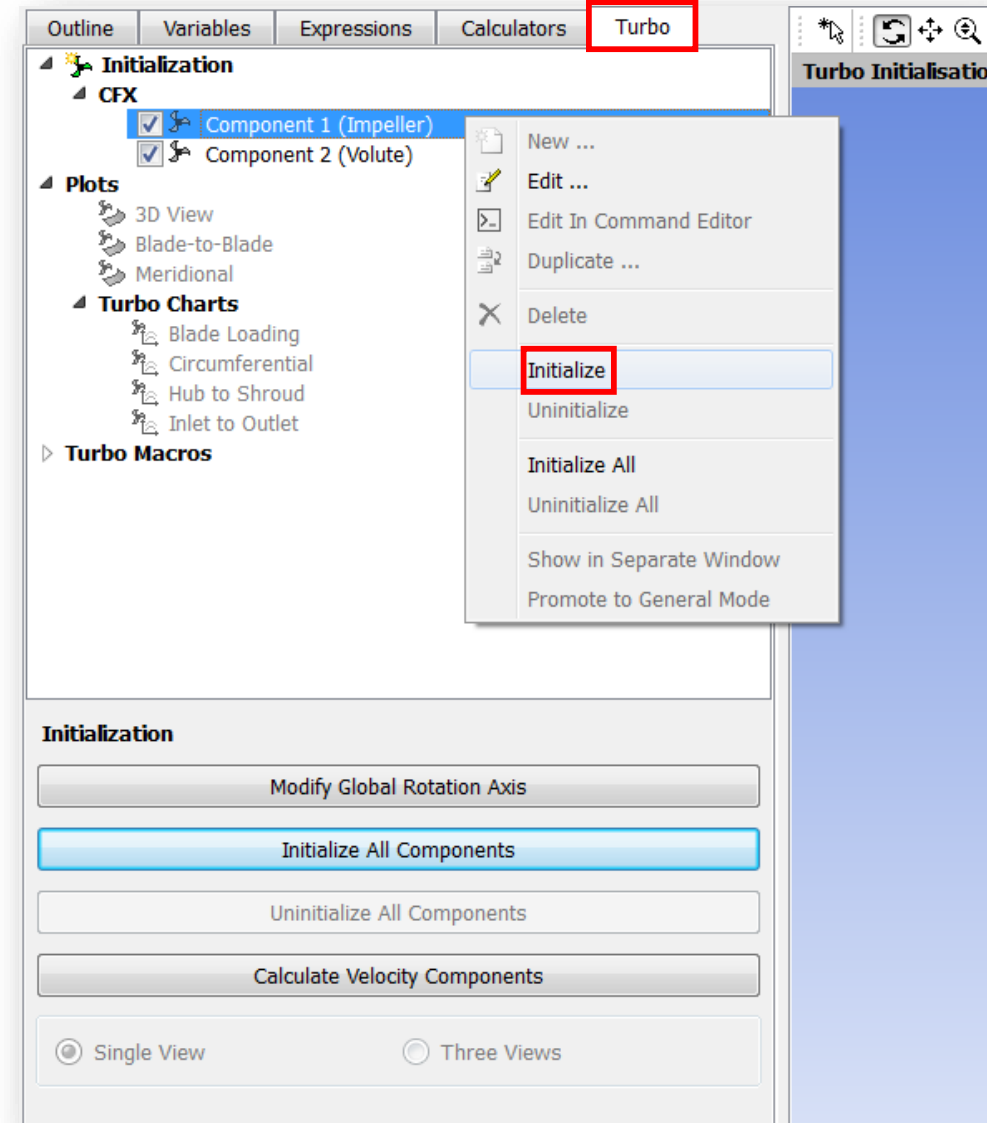
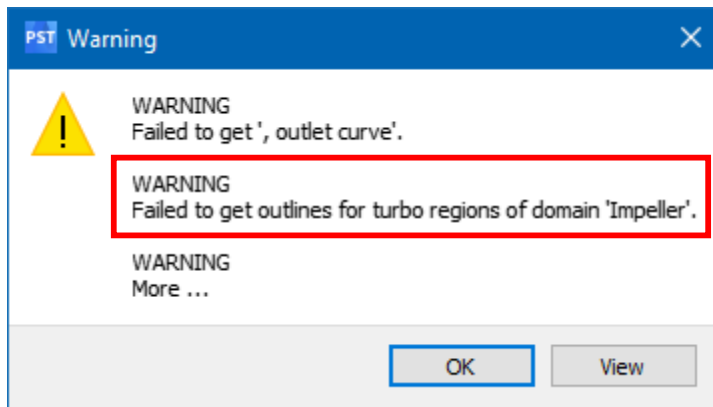
Post-Processing

- Double click on the *Results* Tab in Workbench to launch *CFD-Post*



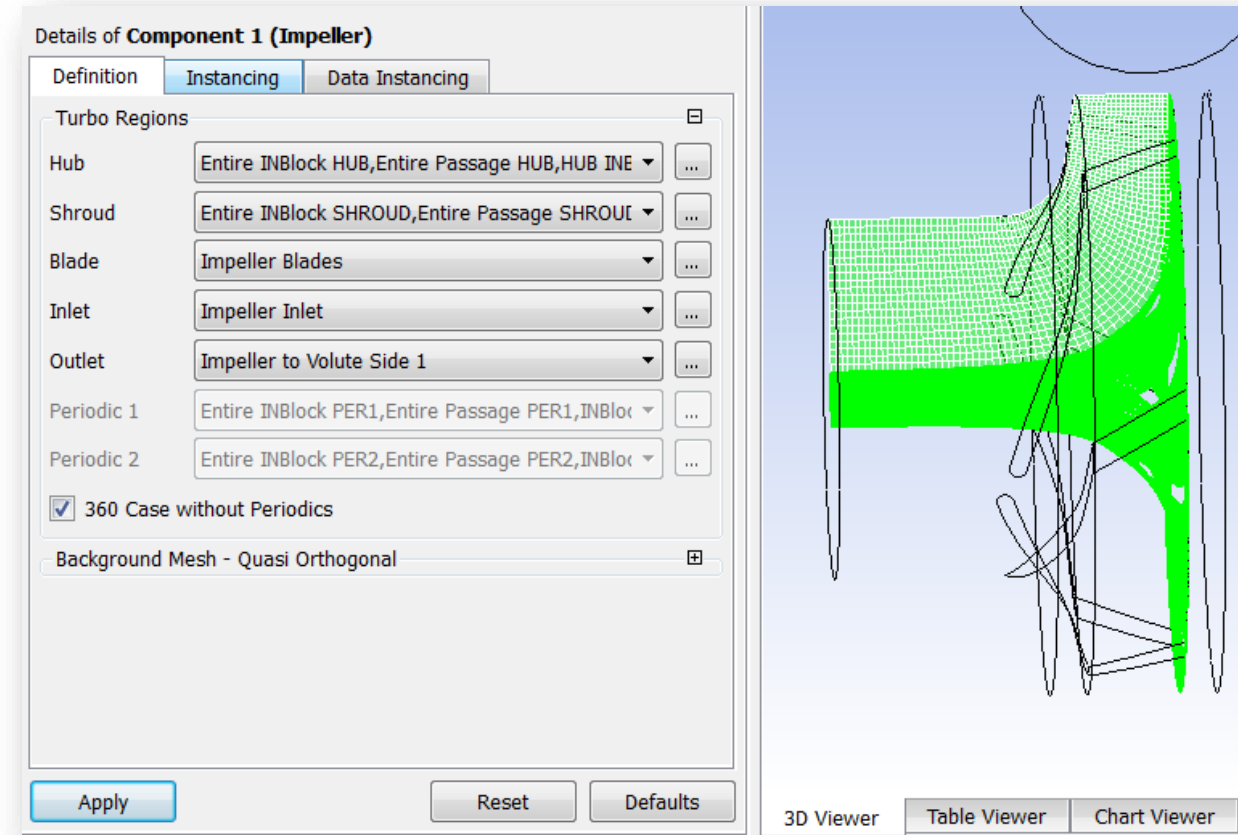
Post-Processing

- We will initialize only the *Impeller* component on the *Turbo* Tab
 - Click on the *Turbo* Tab
 - If prompted to initialize all components, select no
 - Right click on *Component 1 (Impeller)* and select *Initialize*
 - You should receive a message stating *Failed to get outlines for turbo regions...*



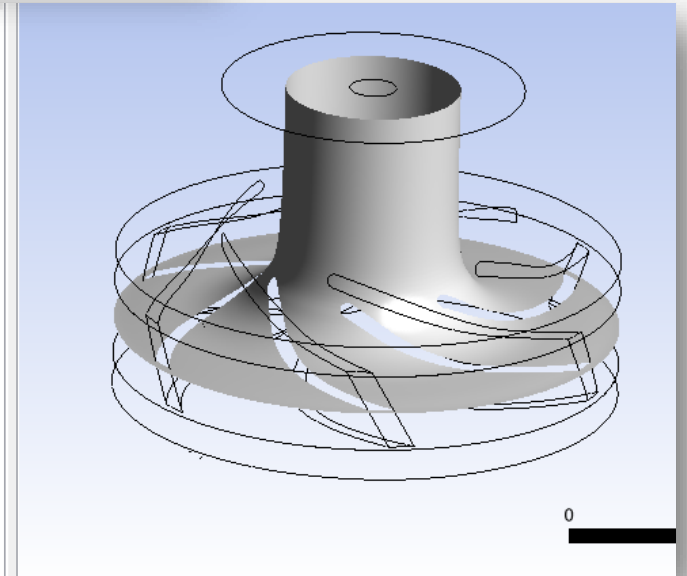
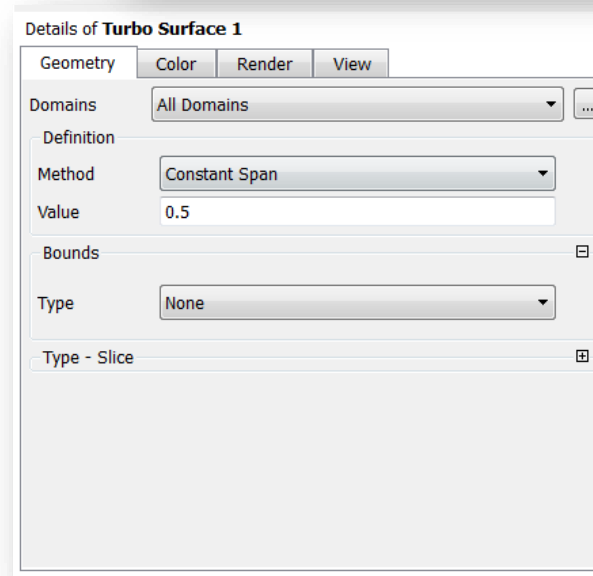
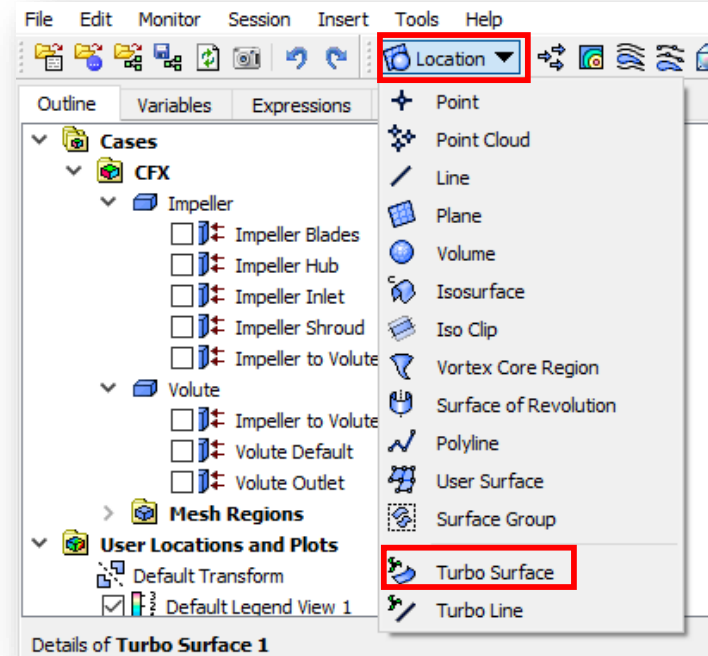
Post-Processing

- To resolve the error, double click on *Component 1*
 - Check the checkbox for *360 Case without Periodics* and click *Initialize*
 - The case should be properly initialized, indicated by the green background mesh



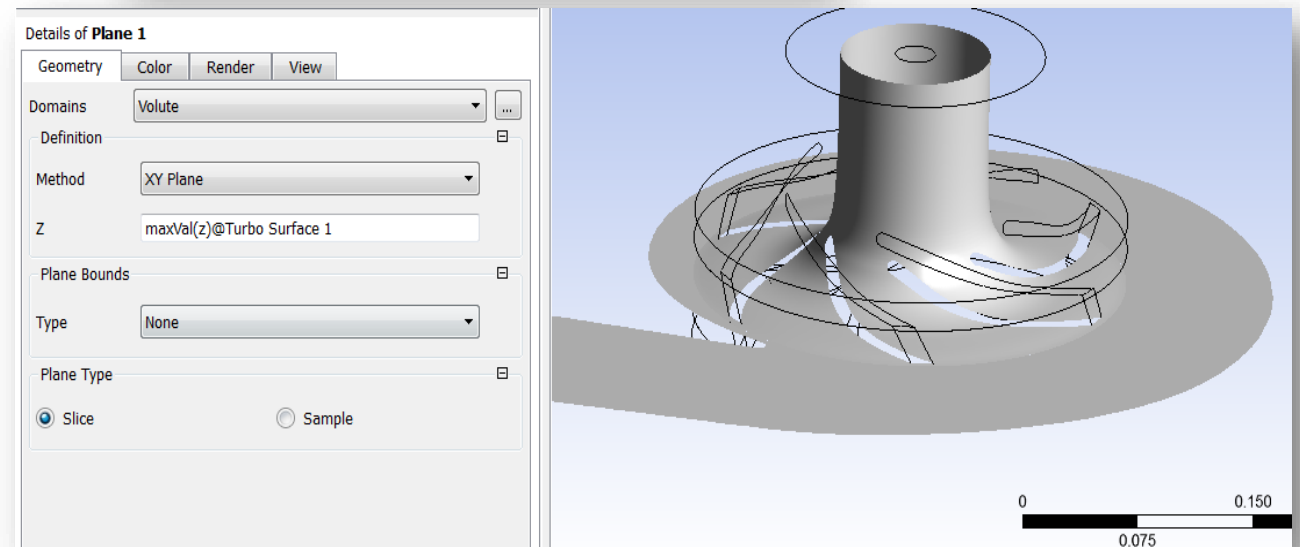
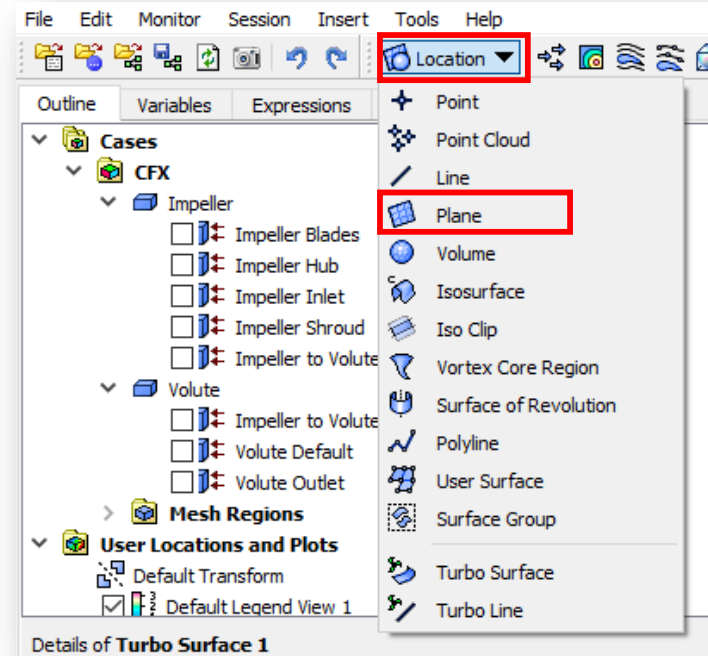
Post-Processing

- Back on the *Outline* Tab, we will create a *Turbosurface* at 50% span
- In the toolbar, from the *Location* drop down select *Turbo Surface*
- In the *details of Turbo Surface 1*
 - *Constant Span*, 0.5
 - *Apply*



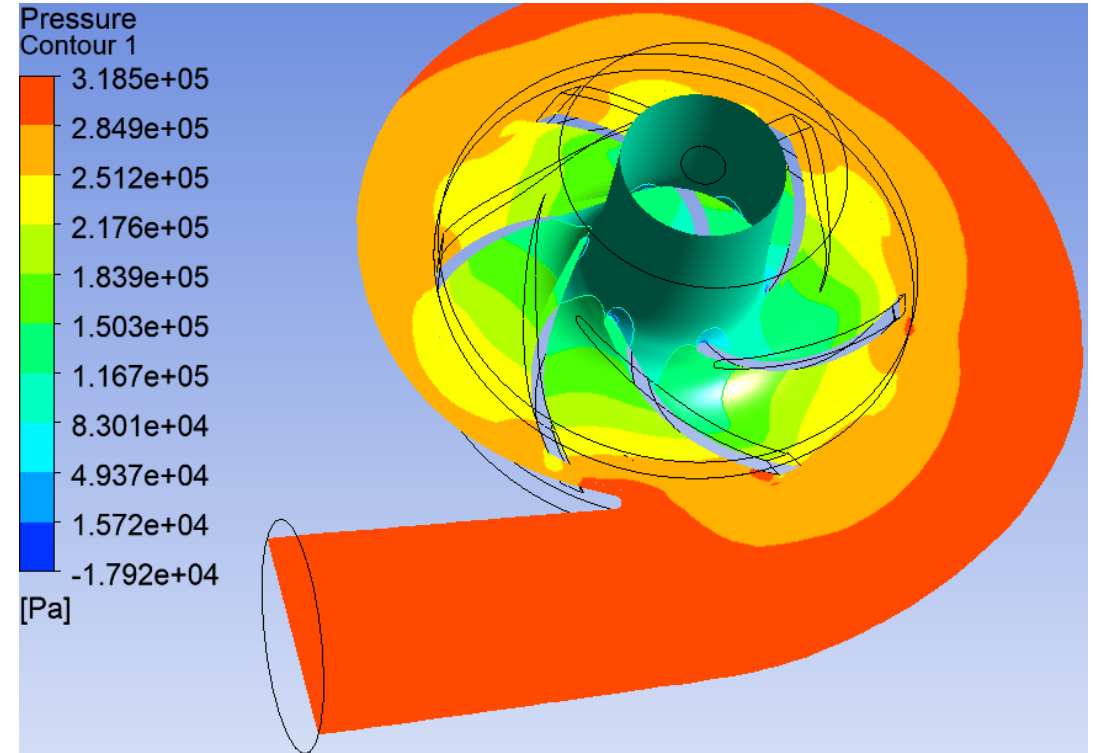
Post-Processing

- We will make a plane that is tangent to this in the volute domain
- In the toolbar, from the *Location* drop down select *Plane*
- In the *details of Turbo Surface 1*
 - *Domains: Volute*
 - *Method: XY Plane*
 - For Z, enter expression:
 $\text{maxVal}(z)@Turbo\ Surface\ 1$
 - *Apply*



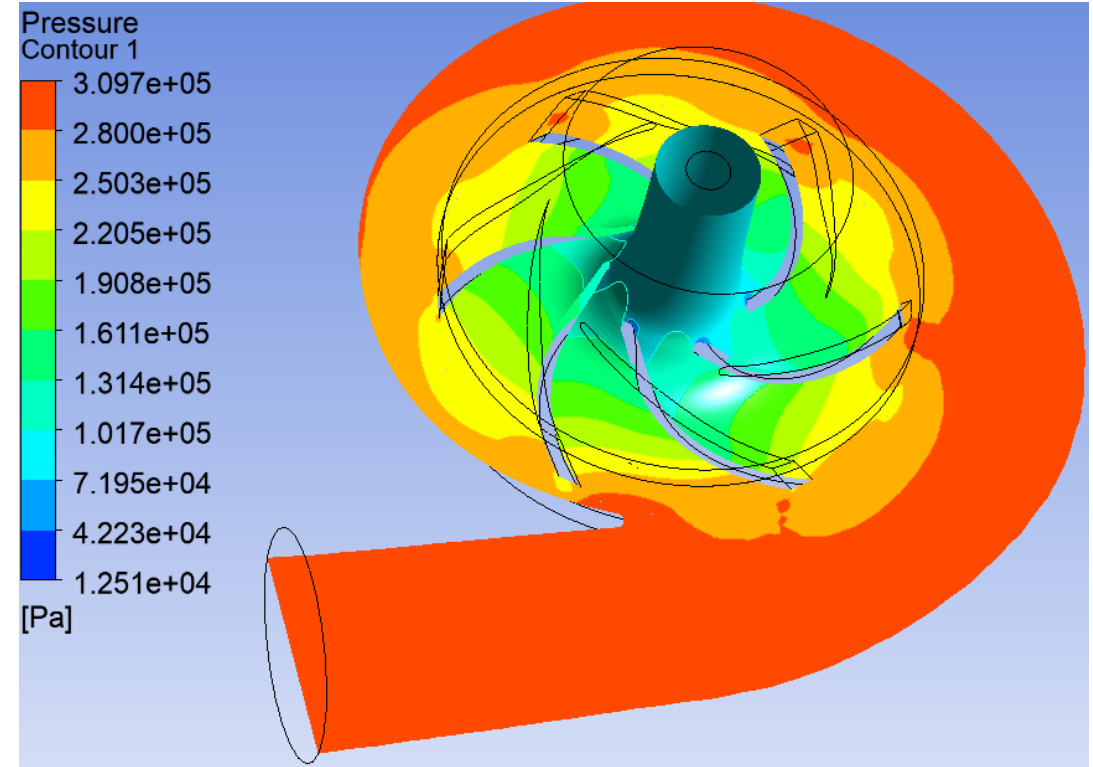
Post-Processing

- Make a contour plot of pressure on both Plane 1 and the Turbo Surface
 - Turn off the visibility of *Plane 1* and *TurboSurface 1*
 - *Insert > Contour*, accept default name
 - For *Locations*, use the three dots icon ... to select both *Plane 1* and *TurboSurface 1*
 - Set the *Range* to *Local*
 - *Apply*



Post-Processing

- Modify *Turbo Surface 1* to visualize at 25% span
 - Double click *Turbo Surface 1*
 - Set *Value* to 0.25
 - *Apply*
 - Note that the *Plane 1* also updates position
- Note the flow non-uniformity downstream of the blades due to the use of a frozen rotor interface in this case



Post-Processing

- Make a table for the Expressions created:
 - Go to the *Table Viewer* Tab
 - Select *New Table* from the toolbar
 - In cell A1, enter the text *Head*
 - In cell B1, enter *=myHead* (or alternatively after entering “=“ *RMB > Expressions* and selected expression *myHead*)
 - In the same way, enter the texts as shown in column A and their corresponding expressions (created on slides 14, 21 and 24) in column B for Mass Flow Rate, Rotation Speed, Torque, Efficiency and Shaft Power
- Note: In column B multiply the expression *myEfficiency* by *1 [rad]* and divide *myPower* by *1[rad]* for proper units

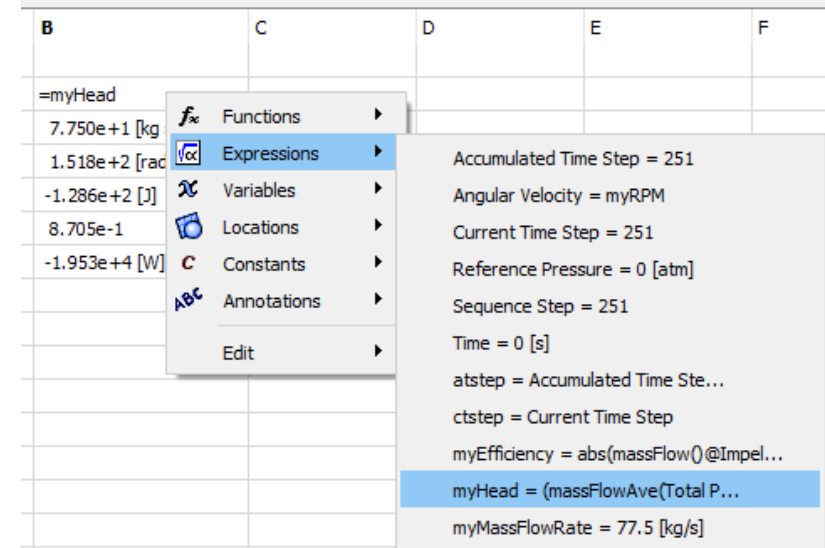


Table 1

	B7	=myPower/1[rad]
	A	B
1		
2	Head	2.237e+1 [m]
3	Mass Flow Rate	7.750e+1 [kg s ⁻¹]
4	Rotation Speed	1.518e+2 [radian s ⁻¹]
5	Torque	-1.286e+2 [J]
6	Efficiency	8.705e-1
7	Shaft Power	=myPower/1[rad]

Post-Processing

- On your own:
- Create a *Vector* plot on *Plane 1* and *Turbo Surface 1*, and try plotting both *Velocity* and *Velocity in Stn Frame*
- On the *Turbo Tab > Initialization*, click *Calculate Velocity components*. Then try to calculate the average flow angle at the blade leading edge and trailing edge

Summary

- In this workshop, you have learned the following:
 - Combining meshes for different components in CFX Pre
 - Duplicating single passage meshes for full 360
 - Setting up multiple frames of reference with a domain interface
 - Setting up monitor points for quantities of interest
 - Determining convergence of a CFX solution
 - Post processing a mix of rotating and stationary components