

Reg. No		

## **B.Tech Degree Examination, DECEMBER 2023**

Fourth Semester

## 18MEC107T - APPLIED THERMAL ENGINEERING

(For the candidates admitted during the academic year 2018-19 to 2021-22)

OPEN BOOK EXAMINATION

18MECIOTTO

Note:

i. Specific approved THREE text books (Printed or photocopy) recommended for the course. ii. Handwritten class notes (certified by the faculty handling the course / Head of the Department).

Time: 3 Hours			Max. I	s: 100	
	Answer <b>FIVE</b> (Question No. 1 is		Mark	s BL	CO
	An air standard limited pressure cycle has a compression ratio of 15 and compression begins at 0.1MPa, 313K. The maximum pressure is limited to 6MPa and the heat added is 1.675MJ/kg. Compute a) the heat supplied at constant volume per kg of air, b) the heat supplied at constant pressure per kg of air, c)the work done per kg of air, d) the cycle efficiency, e)the temperature at the end of the constant volume heating process, f) the cut-off ratio and g) the mep of the cycle. (Take γ=1.4, Cp = 1.005 kJ/kg K, R= 0.287 kJ/kg K)			3	1
b , jil	regeneration	cycle? (B) net work output increases because of regeneration (D) regeneration is not advisable for Brayton cycle	1	1	1
C. jú.		erature of the Otto, Diesel and Dual air (B) Heat supplied (D) Heat rejected	1	1	1
2 i.	if The gravimetric analysis of coal gives 79% of carbon, 9% of hydrogen, 4% of moisture and 8% of ash. Actual air supplied is 19 kg per kg of coal. Calculate the theoretical amount of air required, if 80% of carbon is burned to CO <sub>2</sub> and the remaining to CO. Also determine the volumetric composition of dry products of combustion		18	5	2
<b>b</b> . ii.	place	mbustion ensures that (B) incomplete combustion of fuel occurs (D) air is used for combustion	1	1	2
C.jh.	The calorific value determined by the bomb ca (A) lower calorific value at constant pressure	lorimeter is (B) higher calorific value at constant pressure (D) higher calorific value at constant volume	1	1	2

3 In a test of an oil engine under full load condition the following results were obtained. Indicated Power = 32 kWBrake power = 28 kW Fuel used = 7 kg/h Rate of flow of water through gas calorimeter = 14 kg/min Cooling water flow rate = 9 kg/min Calorific value of fuel = 43 MJ/kg Inlet temperature of cooling water =  $20^{\circ}$ C Outlet temperature of cooling water = 45°C Inlet temperature of water to exhaust gas calorimeter =  $20^{\circ}$ C Outlet temperature of water to exhaust gas calorimeter = 75°C Final temperature of the exhaust gases = 210°C Room temperature =  $24^{\circ}$ C Air-fuel ratio on mass basis = 21Mean specific heat of exhaust gas = 1 kJ/kg K Specific heat of water = 4.18 J/kg KDraw up a heat balance sheet in kJ/hr. Estimate the specific fuel consumption in g/kW-hr and Thermal and Mechanical efficiencies. The bore and stroke of a single cylinder four-stroke engine are 100 mm and 160 mm respectively. If the brake torque is 50 Nm, the bmep is (A) 15 bar (B) 10 bar (C) 5 bar (D) 7.6 bar iid. Volumetric efficiency is a measure of (A) speed of the engine (B) power of the engine (C) breathing capacity of the engine (D) pressure rise in the cylinder i. A single-acting, two-stage reciprocating compressor with complete intercooling delivers 12 kg per minute of air at 16 bar. The suction occurs at 1.01 bar and 27°C. The compression and expansion processes are reversible with polytrophic index n = 1.3. The compressor runs at 500 rpm. Assuming the density of air as 1.2 kg/m<sup>3</sup>, Calculate the (a) The power required to drive the compressor. (b) The isothermal efficiency. (c) The free air delivered. (d) The heat transferred in intercooler. (e) The swept and clearance volumes for each cylinder if the clearance ratios for L.P. and H.P. cylinders are 0.04 and 0.06, respectively. 1 Po PI The above pV diagram is for (A) Roots blower (B) Vanes blower (C) Centrifugal compressor (D) Reciprocating compressor Roots blower is an example of 1 1 (A) Reciprocating (positive displacement) (B) Rotary (positive displacement) compressor compressor (C) Centrifugal compressor (D) Axial flow compressor

5	A food storage locker requires a refrigeration temperature of -5°C and a condenser tempera cooled 6°C before entering the expansion vertical before leaving the evaporator coil. The consideration adiabatic. A two-cylinder vertical single acting the bore is to be used operating at 900 rpm. Do (a) the refrigerating effect/kg, (b) the mass of refrigerant to be circulated per (c) the theoretical piston displacement per min (d) the theoretical power, (e) the co-efficient of performance, (f) The heat removed through condenser/kg and (g) The theoretical bore and stroke of compressions.	ature of 40°C. The refrigerant, R-12, is subvalve, and the vapour is superheated 5°C impression of the refrigerant is reversible in compressor with stroke equal to 1.5 times etermine immute, mute,	18	5	5	
b.ji!	A Carnot refrigerator requires 70 kJ/min of v -40°C. The COP of this refrigerator is (A) 4 (C) 5	(B) 3 (D) 1/70	1	1	5	
c . ijh.	Name the Psychometric processes involved in cities like Chennai, Mumbai etc (A) (a) Cooling and humidification (C) (c) Heating and humidification	(B) (b) Cooling and dehumidification (D) (d) Heating and dehumidification	1	1	5	,
6 jl.	During the trial of a single-cylinder, four-strobtained.  Cylinder diameter = 200 mm  Stroke = 400 mm  Mean effective pressure = 6 bar  Torque = 407 Nm  Speed = 250 rpm  Oil consumption = 4 kg/h  Calorific value of fuel = 43 MJ/kg  Cooling water flow rate = 4.5 kg/min  Air used per kg of fuel = 30 kg  Rise in cooling water temperature = 45°C  Temperature of exhaust gases = 420°C  Room temperature = 20°C  Mean specific heat of exhaust gas = 1 kJ/kg K  Specific heat of water = 4.18 kJ/kg K  Find the IP, BP and draw up a heat balance sh	roke oil engine, the following results were	18	5	3	
o, id.	If N is the rpm, number of power strokes/min (A) 2N (C) N	in a four-stroke engine is (B) N/2 (D) 4N	1	1	•	3
C. iii.	Indicated power is directly proportional to (A) torque (C) cylinder peak pressure	<ul><li>(B) air consumption</li><li>(D) none of the above</li></ul>	1	1	3	3
7 /t. <b>a.i)</b>	a) In a simple vapour compression cycle, fol R-12 at various points. Compressor inlet: $h_1 = 183.2 \text{ kJ/kg}$ , $v_1 = 0.222.6 \text{ kJ/kg}$ , $v_2 = 0.0164 \text{ m}^3/\text{kg}$ Condenser exit: $h_3 = 84.9 \text{ kJ/kg}$ , $v_3 = 0.0083  the compressor is 1.5 litres per stroke and its efficiency is 80%. The speed of the compress the compressor (kW), (b) the refrigerating efficiency$	$0.0767 \text{ m}^3/\text{kg}$ , Compressor discharge: $h_2 = \text{m}^3/\text{kg}$ , The piston displacement volume for volumetric for is 1600 rpm. Find (a) the power rating of	10	1		5
ii),	ompression cycle, the refrigerant enters the cleaves with an enthalpy of 183 kJ/kg. The er is 210 kJ/kg. Show the cycle on T-s and p-h (h) power input to compressor, and (c) rate of	evaporator with an enthalpy of 75 kJ/kg and athalpy of the refrigerant after compression diagrams. Calculate the following: (a) COP,	8	1		

ii. One tonne refrigerating machine means that (A) one tonne is the total mass of the

machine

(C) one tonne of water can be converted into ice

(B) one tonne of refrigerant is used

(D) one tonne of ice when melts from and at 0°C in 24 hours, the refrigeration effect produced is equivalent to 210 kJ/min.

ii. In an ideal vapour compression refrigeration cycle, which process is irreversible?
 (A) Compression
 (B) Heat rejection

(C) Throttling

(D) Heat absorption

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