

Outline

- ▶ Reynolds transport theorem
- ▶ Conservation of mass
- ▶ Euler's axioms
- ▶ Equations of motion

Governing equations in mechanics

- ▶ The equations
 - ▶ Conservation of mass
 - ▶ Equations of motion
 - ▶ Energy equation
- ▶ Formulated for material bodies in motion (i.e. $V(t)$)
- ▶ In fluid mechanics it is more convenient to work with control volumes, i.e. a volume independent of time ($V=\text{const}$)
- ▶ Reynolds transport theorem is called for

Extensive and intensive properties

- ▶ Extensive property
 - ▶ A physical property
 - ▶ A function of the volume or mass of a body
- ▶ Intensive property
 - ▶ Physical property independent of body volume or mass
 - ▶ Density when given per unit volume
 - ▶ Specific property when given per unit mass
- ▶ Examples
 - ▶ Linear momentum: $\mathbf{p} = \int_V \mathbf{v} \rho dV$
 - ▶ Kinetic energy: $K = \int_V \frac{\mathbf{v} \cdot \mathbf{v}}{2} \rho dV$
 - ▶ Internal energy: $E = \int_V \epsilon \rho dV$
- ▶ General form for *extensive quantity*: $B(t) = \int_{V(t)} \beta \rho dV$
- ▶ Specific intensive property $\beta(\mathbf{r}, t)$
- ▶ $\beta(\mathbf{r}, t)$ is a property per unit mass

Derivation Reynolds Transport Theorem

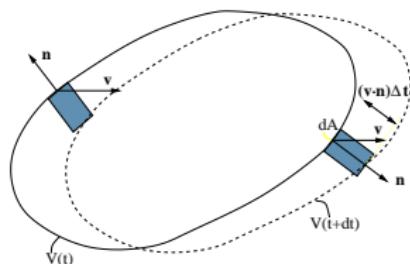
- ▶ Density b of a property B :

$$B(t) = \int_{V(t)} b(\mathbf{r}, t) dV$$

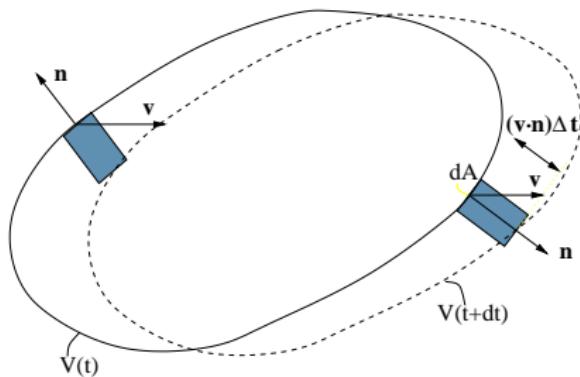
- ▶ Canonical definition of a derivative

$$\dot{B} = \lim_{\Delta t \rightarrow 0} \frac{B(t + \Delta t) - B(t)}{\Delta t}$$

- ▶ A closer look at $B(t + \Delta t)$



A closer look at $B(t + \Delta t)$



$$\begin{aligned} B(t + \Delta t) &= \int_{V(t + \Delta t)} b(\mathbf{r}, t + \Delta t) dV \\ &= \int_{V(t)} b(\mathbf{r}, t + \Delta t) dV + \int_{\Delta V} b(\mathbf{r}, t + \Delta t) dV \\ \int_{\Delta V} b(\mathbf{r}, t + \Delta t) dV &= \int_{A(t)} b(\mathbf{r}, t + \Delta t) (\mathbf{v} \cdot \mathbf{n} \Delta t) dA \end{aligned}$$

A closer look at $B(t + \Delta t)$ (cont)

By collection terms:

$$\begin{aligned}\frac{B(t + \Delta t) - B(t)}{\Delta t} &= \int_{V(t)} \frac{b(\mathbf{r}, t + \Delta t) - b(\mathbf{r}, t)}{\Delta t} dV \\ &\quad + \int_{A(t)} b(\mathbf{r}, t + \Delta t) (\mathbf{v} \cdot \mathbf{n}) dA\end{aligned}$$

By letting $\Delta t \rightarrow 0$ the Reynolds Theorem is obtained

$$\dot{B} = \int_{V(t)} \frac{\partial b}{\partial t} dV + \int_{A(t)} b (\mathbf{v} \cdot \mathbf{n}) dA$$

Reynolds Transport Theorem

$$\dot{B} = \int_{V(t)} \frac{\partial b}{\partial t} dV + \int_{A(t)} b(\mathbf{v} \cdot \mathbf{n}) dA$$

- ▶ As the derivative is inside the integral sign we may now assume
 $V(t) = V = \text{constant}$
- ▶ An equivalent representation is therefore

$$\dot{B} = \frac{d}{dt} \int_V b dV + \int_A b(\mathbf{v} \cdot \mathbf{n}) dA$$

Material derivative of an extensive property

- ▶ The governing equations contain material derivatives of extensive properties
- ▶ It may be shown based on conservation of mass that

$$\dot{B} = \frac{d}{dt} \int_{V(t)} \beta \rho dV = \int_{V(t)} \dot{\beta} \rho dV$$

- ▶ Transform equations to apply for a *fixed region in space* in fluid mechanics for convenience
- ▶ A *control volume* is such a fixed region

Conservation of mass

- ▶ The conservation of mass equation is obtained by setting $b = \rho$ in Reynolds Transport Theorem

$$\dot{m} = \frac{d}{dt} \int_{V(t)} \rho(\mathbf{r}, t) dV = \int_V \frac{\partial \rho}{\partial t} dV + \int_A \rho (\mathbf{v} \cdot \mathbf{n}) dA = 0$$

- ▶ By using the Gauss' theorem for the surface integral:

$$\dot{m} = \int_V \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) dV = 0$$

Differential form for mass conservation

- ▶ The integral form (control volume) formulation

$$\dot{m} = \int_V \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) dV = 0$$

- ▶ Must be valid for arbitrary V thus the integrand must be zero

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

- ▶ On component form

$$\frac{\partial \rho}{\partial t} + (\rho v_i)_{,i} = 0$$

Mass conservation for incompressible flow

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

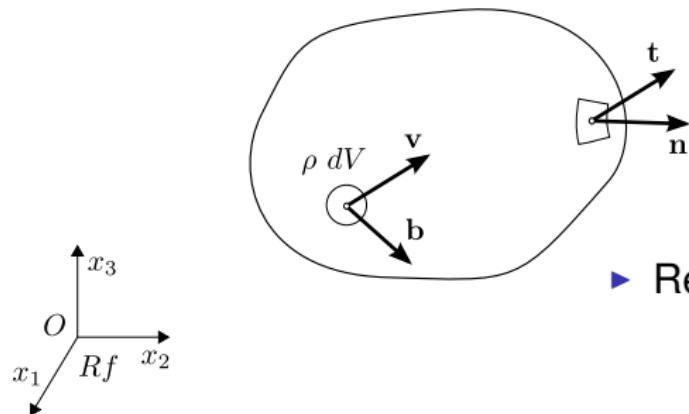
- ▶ $\rho = \text{constant}$

$$\nabla \cdot \mathbf{v} = 0 \quad \Leftrightarrow \quad v_{i,i} = 0$$

Equations of motion

- ▶ Newton's laws are valid for the motion of a mass particle
- ▶ Cannot directly be transferred to a body of continuously distributed matter
- ▶ Two axioms are postulated in Continuum Mechanics
 - ▶ Euler's first axiom: Law of balance of linear momentum
 - ▶ Euler's second axiom: Law of balance of angular momentum

Forces and moments



► Resultant force

$$\mathbf{f} = \int_A \mathbf{t} dA + \int_V \mathbf{b} \rho dV$$

- Body forces \mathbf{b} per unit mass
- Contact force \mathbf{t} per unit area
- Particle velocity \mathbf{v}

► Resultant moment \mathbf{m}_O

$$\mathbf{m}_O = \int_A \mathbf{r} \times \mathbf{t} dA + \int_V \mathbf{r} \times \mathbf{b} \rho dV$$

Law of balance of linear momentum

- ▶ Linear momentum $\mathbf{p} = \int_V \mathbf{v} \rho dV$
- ▶ Euler's first axiom: *The rate of change of the linear momentum for a continuum volume equals the resultant force*

$$\dot{\mathbf{p}} \equiv \int_V \dot{\mathbf{v}} \rho dV = \mathbf{f}$$

- ▶ Equivalent form

$$\int_V \dot{\mathbf{v}} \rho dV = \int_A \mathbf{t} dA + \int_V \mathbf{b} \rho dV$$

Relation to Newton's second law

- ▶ Center of mass $\mathbf{r}_C = \frac{1}{m} \int_V \mathbf{r} \rho dV$
- ▶ Velocity and acceleration of the center of mass

$$\mathbf{v}_C = \dot{\mathbf{r}}_C = \frac{1}{m} \int_V \dot{\mathbf{r}} \rho dV = \frac{1}{m} \int_V \mathbf{v} \rho dV$$

$$\mathbf{a}_C = \dot{\mathbf{v}}_C = \ddot{\mathbf{r}}_C = \frac{1}{m} \int_V \dot{\mathbf{v}} \rho dV = \frac{1}{m} \int_V \mathbf{a} \rho dV$$

- ▶ Linear momentum: $\mathbf{p} = \int_V \mathbf{v} \rho dV = m\mathbf{v}_C$
- ▶ Euler's first axiom yields Newton's second law for the center of mass

$$\mathbf{f} = m\mathbf{a}_C$$

Law of balance of angular momentum

- ▶ Angular momentum $\mathbf{I}_O = \int_V \mathbf{r} \times \mathbf{v}_\rho dV$
- ▶ Euler's second axiom: *The rate of change of the angular momentum for a continuum volume equals the resultant moment*

$$\dot{\mathbf{I}}_O = \int_V \mathbf{r} \times \dot{\mathbf{v}}_\rho dV$$

- ▶ Equivalent form:

$$\int_V \mathbf{r} \times \dot{\mathbf{v}}_\rho dV = \int_A \mathbf{r} \times \mathbf{t} dA + \int_V \mathbf{r} \times \mathbf{b}_\rho dV$$

Vector relation for simplification in angular momentum expression

- ▶ Outset

$$\dot{\mathbf{r}} \times \mathbf{v} = \dot{\mathbf{r}} \times \mathbf{v} + \mathbf{r} \times \dot{\mathbf{v}}$$

- ▶ We have

$$\dot{\mathbf{r}} = \mathbf{v} \quad \text{and} \quad \mathbf{v} \times \mathbf{v} = 0$$

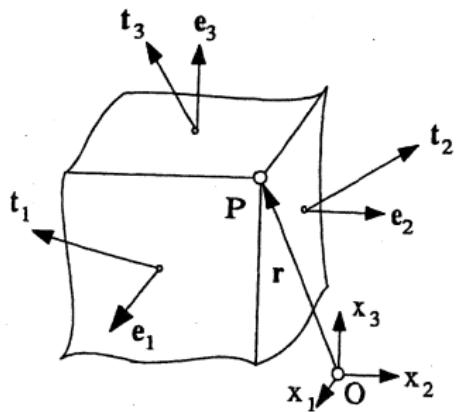
- ▶ Thus

$$\dot{\mathbf{r}} \times \mathbf{v} = \mathbf{r} \times \dot{\mathbf{v}} = \mathbf{r} \times \mathbf{a}$$

Coordinate stresses

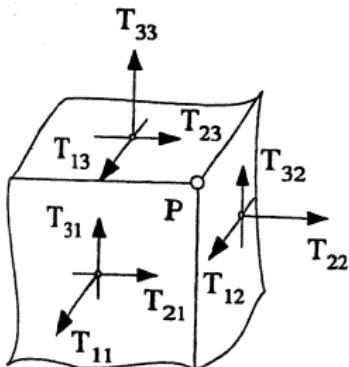
- ▶ Stress \mathbf{t}_k on surfaces with unit normals \mathbf{e}_k
- ▶ Components of the stress vectors are denoted T_{ik}

$$\mathbf{t}_k = T_{ik} \mathbf{e}_i \quad \Leftrightarrow \quad \mathbf{e}_i \cdot \mathbf{t}_k = T_{ik}$$



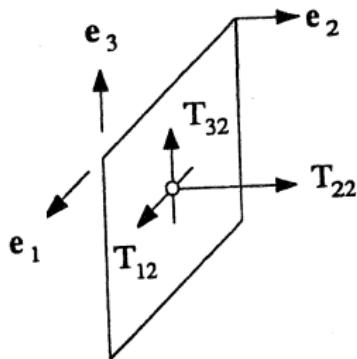
$$\mathbf{t}_k = T_{ik} \mathbf{e}_i$$

=



Coordinate stresses

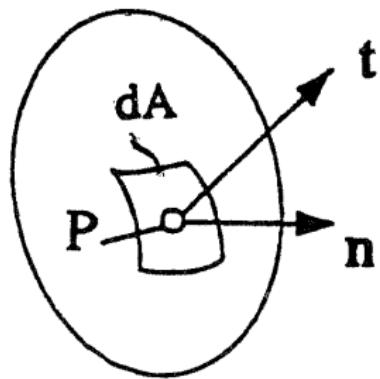
- ▶ T_{ik} is the coordinate stresses in the particle P
- ▶ T_{ik} are elements of the stress matrix T in P with respect to Ox
- ▶ Normal stresses: T_{11}, T_{22}, T_{33}
- ▶ Shear stresses: T_{ik} , when $i \neq k$
- ▶ A positive coordinate stress acts in the direction of the positive coordinate axis on that side of the surface facing the positive direction of a coordinate axis



Cauchy's stress theorem and the stress tensor

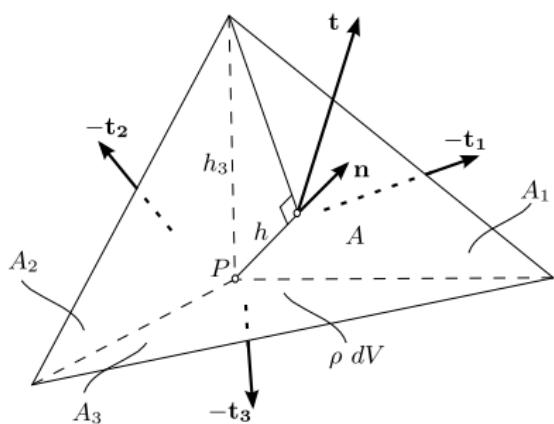
- ▶ Consider an arbitrary plane through P
- ▶ Normal vector $\mathbf{n} = n_k \mathbf{e}_k$
- ▶ Stress vector $\mathbf{t} = t_i \mathbf{e}_i$
- ▶ Coordinate stresses T_{ik}
- ▶ Cauchy's stress theorem

$$t_i = T_{ik} n_k \Leftrightarrow \mathbf{t} = \mathbf{T}\mathbf{n}$$



Proof of Cauchy's stress theorem

- ▶ Balance of linear momentum

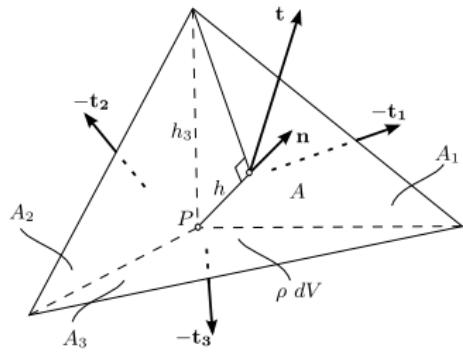


$$\int_V \mathbf{a}\rho dV = \sum_{k=1}^3 \int_{A_k} -\mathbf{t}_k dA + \int_A \mathbf{t} dA + \int_V \mathbf{b}\rho dV$$

- ▶ Let \mathbf{t}_k , \mathbf{t} , $\mathbf{b}\rho$ and $\mathbf{a}\rho$ represent mean values

$$-\mathbf{t}_k A_k + \mathbf{t} A + \mathbf{b}\rho V = \mathbf{a}\rho V$$

Geometric relations



- ▶ h_k denote edges \parallel to base vectors \mathbf{e}_k
- ▶ As \mathbf{n} is a unit vector: $n_k = \frac{h}{h_k}$
- ▶ The volume V may be expressed

$$V = \frac{1}{3} Ah = \frac{1}{3} A_1 h_1 = \frac{1}{3} A_2 h_2 = \frac{1}{3} A_3 h_3$$

- ▶ Which together yields:
 $A_k = An_k$ and $V = \frac{1}{3} Ah$

Substitution in to averaged equation

- ▶ Substitution of $A_k = An_k$ and $V = \frac{1}{3}Ah$

$$-\mathbf{t}_k A_k + \mathbf{t}A + \mathbf{b}\rho V = \mathbf{a}\rho V$$

$$-\mathbf{t}_k n_k + \mathbf{t} + \mathbf{b}\rho \frac{h}{3} = \mathbf{a}\rho \frac{h}{3}$$

- ▶ Let $h \rightarrow 0 \Rightarrow \mathbf{t} = \mathbf{t}_k n_k$
- ▶ From before $\mathbf{t}_k = T_{ik} \mathbf{e}_i$
- ▶ Which yields

$$\mathbf{t} = T_{ik} n_k \mathbf{e}_i \Leftrightarrow t_i = T_{ik} n_k$$

Cauchy's stress theorem

- ▶ The stress vector \mathbf{t} on a surface is uniquely determined by the stress tensor \mathbf{T} and the unit normal \mathbf{n}

$$\mathbf{t} = \mathbf{T} \cdot \mathbf{n}$$

- ▶ Normal stress $\sigma = \mathbf{n} \cdot \mathbf{t} = \mathbf{n} \cdot \mathbf{T} \cdot \mathbf{n} = n_i T_{ik} n_k$
- ▶ Shear stress $\tau = |\mathbf{n} \times \mathbf{t}| = \sqrt{\mathbf{t} \cdot \mathbf{t} - \sigma^2}$
- ▶ Stress tensor T and its components in a general Cartesian coordinate system Ox

$$T_{ik} = \mathbf{e}_i \cdot \mathbf{T} \cdot \mathbf{e}_k$$

Cauchy equations of motion

- ▶ Euler's axioms are equations of motion for a *body* of continuous material
- ▶ From these laws we derive field equations valid for *particles* in a continuum

Derivation of Cauchy's equations of motion

- ▶ The balance of linear momentum

$$\int_V \dot{\mathbf{v}} \rho dV = \int_A \mathbf{t} dA + \int_V \mathbf{b} \rho dV$$

- ▶ Cauchy's stress theorem $\mathbf{t} = \mathbf{T} \cdot \mathbf{n}$ and Gauss' integration theorem

$$\int_V \dot{\mathbf{v}} \rho dV = \int_V \nabla \cdot \mathbf{T} + \mathbf{b} \rho dV$$

- ▶ Integrand must be zero for validity for arbitrary V

$$\rho a_i = T_{ik,k} + \rho b_i$$

Symmetry of stress tensor

- ▶ From balance of angular/linear momentum one may show:

$$T_{ij} = T_{ji} \quad \Leftrightarrow \quad \mathbf{T}^T = \mathbf{T}$$

Summary

- ▶ Reynolds transport theorem
- ▶ Conservation of mass
- ▶ Euler's axioms
- ▶ Equations of motion