Velocity profiles for straight pipe flow

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Outline

- Nondimensionalized version of the general NS equations
- Nondimensionalized equations for straight pipe flow
- Womersley number
 - Friction dominated (α small)
 - Inertia dominated (α large)
 - Womersley profiles for arbitrary α

Dimensionless Navier-Stokes equations

The incompressible NS equations

$$rac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot
abla) \mathbf{v} = -rac{1}{
ho}
abla
ho + rac{\mu}{
ho}
abla^2 \mathbf{v} + \mathbf{b}$$

Characteristic constant scales

$$\mathbf{x}^* = \mathbf{x}/L, \qquad \mathbf{v}^* = \mathbf{v}/V,$$
 $t^* = t/\theta, \qquad p^* = p/\rho V^2$

Dimensionless Navier-Stokes equations

▶ The incompressible NS equations

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Dimensionless NS-equations

$$Sr\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla)\mathbf{v} = -\frac{1}{\rho}\nabla \rho + \frac{1}{Re}\nabla^2 \mathbf{v} + \frac{1}{Fr^2}\mathbf{b}$$

Dimensionless Navier-Stokes equations

▶ The incompressible NS equations

$$\frac{\partial \mathbf{v}}{\partial t} + (\mathbf{v} \cdot \nabla) \mathbf{v} = -\frac{1}{\rho} \nabla \rho + \frac{\mu}{\rho} \, \nabla^2 \mathbf{v} + \mathbf{b}$$

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Dimensionless NS-equations

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- Strouhal
 - $Sr = \frac{L}{\theta V}$
 - Periodic flows
- Reynolds
 - $Re = \frac{\rho VL}{\mu}$
 - All viscous flows
- Froude

•
$$Fr = \frac{V}{\sqrt{gL}}$$

Free-surface flows

Fully developed flow in straight pipes

Momentum equation

$$\frac{\partial v}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \frac{v}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v}{\partial r} \right)$$

Characteristic scales

▶ Length: $r^* = r/a$, $z^* = z/a$

▶ Time: $t^* = t\omega$

• Velocity: $v^* = v/V$

Fully developed flow in straight pipes

Momentum equation

$$\frac{\partial v}{\partial t} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \frac{\nu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v}{\partial r} \right)$$

Characteristic scales

▶ Length: $r^* = r/a$, $z^* = z/a$

▶ Time: $t^* = t\omega$

• Velocity: $v^* = v/V$

Dimensionless form

$$\left(a^{2}\frac{\omega}{\nu}\right)\frac{\partial v^{*}}{\partial t^{*}} = -\left(\frac{a}{\rho\nu V}\right)\frac{\partial p}{\partial z^{*}} + \frac{1}{r^{*}}\frac{\partial}{\partial r^{*}}\left(r^{*}\frac{\partial v^{*}}{\partial r^{*}}\right)$$

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- Characteristic scales
 - ▶ Length: $r^* = r/a$, $z^* = z/a$
 - ▶ Time: $t^* = t\omega$
 - Velocity: $v^* = v/V$
- Dimensionless form

$$\left(a^{2}\frac{\omega}{\nu}\right)\frac{\partial v^{*}}{\partial t^{*}} = -\left(\frac{a}{\rho\nu V}\right)\frac{\partial p}{\partial z^{*}} + \frac{1}{r^{*}}\frac{\partial}{\partial r^{*}}\left(r^{*}\frac{\partial v^{*}}{\partial r^{*}}\right)$$

• Characteristic pressure $p^* = p/(\rho \nu V/a)$



Dimensionless straight tube equations

$$\alpha^{2} \frac{\partial v^{*}}{\partial t^{*}} = -\frac{\partial p^{*}}{\partial z^{*}} + \frac{1}{r^{*}} \frac{\partial}{\partial r^{*}} \left(r^{*} \frac{\partial v^{*}}{\partial r^{*}} \right)$$

• Womersley parameter $\alpha = \mathbf{a}\sqrt{\frac{\omega}{\nu}}$

Dimensionless straight tube equations

$$\alpha^{2} \frac{\partial v^{*}}{\partial t^{*}} = -\frac{\partial p^{*}}{\partial z^{*}} + \frac{1}{r^{*}} \frac{\partial}{\partial r^{*}} \left(r^{*} \frac{\partial v^{*}}{\partial r^{*}} \right)$$

- Womersley parameter $\alpha = \mathbf{a}\sqrt{rac{\omega}{
 u}}$
- Inertia dominated
 - $\alpha \to \infty$
 - large vessels
 - high frequency
 - ▶ aorta α ≥ 20

Dimensionless straight tube equations

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- Inertia dominated
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 - large vessels
 - high frequency
 - aorta α ≥ 20

- Friction dominated
 - ho $\alpha o 0$
 - small vessels
 - low frequency
 - capillaries $\alpha = 10^{-2}$

Straight tube velocity profiles

Momentum equation

$$\alpha^{2} \frac{\partial v}{\partial t} = -\frac{\partial p}{\partial z} + \frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v}{\partial r} \right)$$

- Remove * for simplicity
- ▶ Linear in $v \Rightarrow$ superposition of harmonics OK



Straight tube velocity profiles

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- Remove * for simplicity
- ▶ Linear in $v \Rightarrow$ superposition of harmonics OK
- ▶ Driving force¹: $\frac{\partial p}{\partial z} = \frac{\partial \hat{p}}{\partial z} e^{it}$
- Velocity $v = \hat{v}e^{it}$



Straight tube velocity profiles

Momentum equation

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- Remove * for simplicity
- Linear in v ⇒ superposition of harmonics OK
- ▶ Driving force¹: $\frac{\partial p}{\partial z} = \frac{\partial \hat{p}}{\partial z} e^{it}$
- Velocity $v = \hat{v}e^{it}$
- By substitution into momentum equation

$$i\alpha^{2}\hat{\mathbf{v}}(\mathbf{r}) = -\frac{\partial\hat{\mathbf{p}}}{\partial\mathbf{z}} + \frac{1}{\mathbf{r}}\frac{\partial}{\partial\mathbf{r}}\left(\mathbf{r}\frac{\partial\hat{\mathbf{v}}}{\partial\mathbf{r}}\right)$$



¹Note time is dimensionless.

Friction dominated straight pipe flow

Small Womersley parameter ($\alpha \rightarrow 0$)

$$\frac{1}{r}\frac{\partial}{\partial r}\left(r\frac{\partial \hat{\mathbf{v}}}{\partial r}\right) = \frac{\partial \hat{\mathbf{p}}}{\partial z}$$

Friction dominated straight pipe flow

Small Womersley parameter $(\alpha \rightarrow 0)$

$$\frac{1}{r}\frac{\partial}{\partial r}\left(r\frac{\partial \hat{\mathbf{v}}}{\partial r}\right) = \frac{\partial \hat{\mathbf{p}}}{\partial z}$$

Solution

$$\hat{\mathbf{v}} = -\frac{1}{4} \frac{\partial \hat{\mathbf{p}}}{\partial \mathbf{z}} \left(1 - r^2 \right)$$

Friction dominated straight pipe flow

Small Womersley parameter $(\alpha \rightarrow 0)$

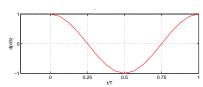
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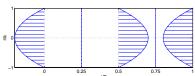
Solution

$$\hat{\mathbf{v}} = -\frac{1}{4} \frac{\partial \hat{\mathbf{p}}}{\partial \mathbf{z}} \left(1 - r^2 \right)$$

► In the time domain

$$v(r,t) = Re\left(-rac{1}{4\mu}rac{\partial
ho}{\partial z}(a^2-r^2)
ight)$$





Inertia dominated straight pipe flow

Large Womersley parameter

$$i\alpha^2 \hat{\mathbf{v}}(\mathbf{r}) = -\frac{\partial \hat{\mathbf{p}}}{\partial \mathbf{z}}$$

Inertia dominated straight pipe flow

Large Womersley parameter

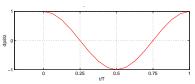
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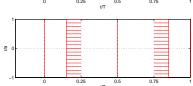
Solution

$$\hat{\mathbf{v}}(\mathbf{r}) = \frac{i}{\alpha^2} \frac{\partial \hat{\mathbf{p}}}{\partial \mathbf{z}}$$

In the time domain

$$v(t) = Re\left(rac{i}{
ho\omega} rac{\partial p}{\partial z}
ight)$$





A first guess on velocity profile for arbitrary α

First guess

$$\frac{\partial \mathbf{v}}{\partial t} = -\frac{\partial \mathbf{p}}{\partial \mathbf{z}} + \frac{\nu}{r} \frac{\partial}{\partial r} \left(r \frac{\partial \mathbf{v}}{\partial r} \right)$$

$$\mathcal{O}(\omega \mathbf{V}) \qquad \qquad \mathcal{O}\left(\frac{\nu \mathbf{V}}{\delta^2} \right)$$

A first guess on velocity profile for arbitrary α

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 Instationary boundary layer thickness

$$\mathcal{O}(\textit{V}\omega) = \mathcal{O}\left(\frac{\nu\textit{V}}{\delta^2}\right) \Rightarrow \delta = \mathcal{O}\left(\sqrt{\frac{\nu}{\omega}}\right)$$

A first guess on velocity profile for arbitrary α

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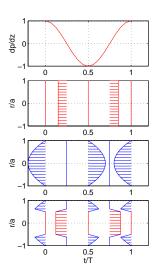
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ightharpoonup Expressed by α

$$\delta = \mathcal{O}\left(\frac{a}{\alpha}\right), \quad \alpha > 1$$



 Momentum equation in frequency domain

$$i\omega\alpha^{2}\hat{\mathbf{v}} = -\frac{\partial\hat{\mathbf{p}}}{\partial z} + \frac{1}{r}\frac{\partial}{\partial r}\left(r\frac{\partial\hat{\mathbf{v}}}{\partial r}\right)$$

 Momentum equation in frequency domain

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• Substitute $s = i^{3/2} \alpha r$

 Momentum equation in frequency domain

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- Substitute $s = i^{3/2} \alpha r$
- ▶ Bessel equation n = 0

$$\frac{\partial^2 \hat{v}}{\partial s^2} + \frac{1}{s} \frac{\partial \hat{v}}{\partial s} + \left(1 - \frac{n^2}{s^2}\right) v = \frac{i}{\rho \omega} \frac{\partial \hat{p}}{\partial z}$$

 Momentum equation in frequency domain

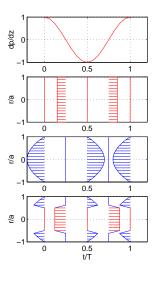
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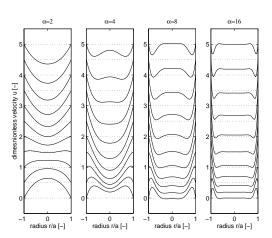
Solution

$$\hat{v}(r) = \frac{i}{\rho\omega} \frac{\partial \hat{p}}{\partial z} \left(1 - \frac{J_0(i^{3/2}\alpha r/a)}{J_0(i^{3/2}\alpha)} \right)$$



Womersley profiles for straight pipe flow

$$v(r,t) = Re\left(rac{i}{
ho\omega} rac{\partial p}{\partial z} \left(1 - rac{J_0(i^{3/2}\alpha r/a)}{J_0(i^{3/2}\alpha)} e^{i\omega t}
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Summary

- Nondimensionalized version of the general NS equations
- Nondimensionalized equations for straight pipe flow
- Womersley number
 - Friction dominated (α small)
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