

EN530.767 - Computational Fluid Dynamics

Assignment 5

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1 Introduction

In this work, solution of Lid Driven Cavity flow was obtained by solving 2-D Navier-Stokes Equation. Lid Driven Cavity flow is considered to be a benchmark problem for validating the fidelity and accuracy of Navier Stokes equations. The results obtained by Ghia et al. [1] was used for validation of results. The schematic of the problem is shown in fig. 1

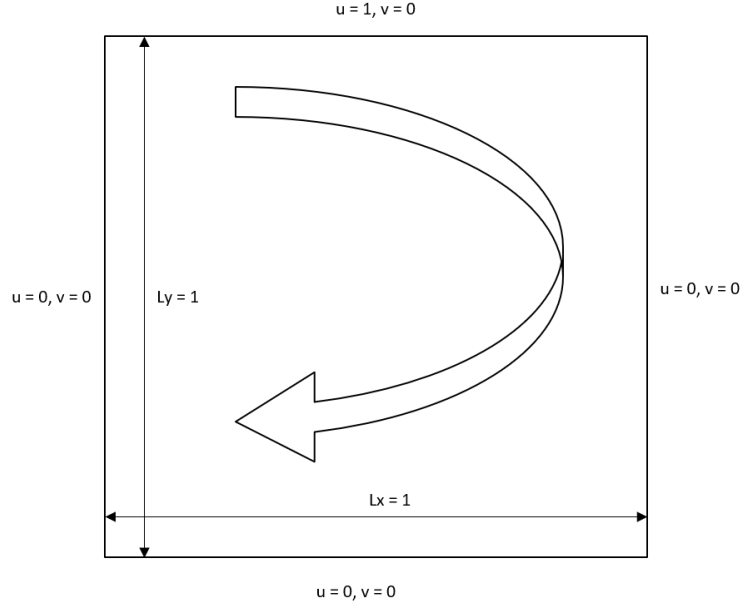


Figure 1: Schematic of Lid Driven Cavity problem

The following sections will elaborate the methods adopted for developing the Navier-Stokes equation solver and the results obtained.

2 Methodology

In this work, non-dimensional 2-D unsteady Navier Stokes equation as shown below was used.

$$\frac{\partial u_i}{\partial x_i} = 0 \quad (1)$$

$$\frac{\partial u_i}{\partial t} + \frac{\partial (U_i u_j)}{\partial x_j} = -\frac{\partial p}{\partial x_j} + \frac{1}{Re} \frac{\partial^2 u_i}{\partial x_i \partial x_j} \quad (2)$$

In order to eliminate the time constraint due to viscous stability criteria the the Diffusion term was given an implicit treatment. Then the equations were discretised using a cell-centered, collocated grid arrangement [2]

[3] where a separate update a staggered mass flux was done on faces using the face velocity U_j which helps in using a compact stencil without going to a staggered mesh approach. The governing equations are then split using fractional step method. Here the Navier-Stokes equations is split into an Advection-Diffusion equation (Eq. 3) and a Pressure Poisson equation (Eq. 4), whose finite difference forms are shown below.

$$\begin{aligned} & \frac{(u^* - u^n)}{\Delta t} + \frac{(Uu)_{i+1,j}^n - (Uu)_{i-1,j}^n}{2\Delta x} + \frac{(Vu)_{i,j+1}^n - (Vu)_{i,j-1}^n}{2\Delta y} \\ &= \frac{1}{Re} \left(\frac{u_{i+1,j}^* - 2u_{i,j}^* + u_{i-1,j}^*}{\Delta x^2} + \frac{u_{i,j+1}^* - 2u_{i,j}^* + u_{i,j-1}^*}{\Delta y^2} \right) \end{aligned} \quad (3)$$

$$\frac{\nabla \cdot u^*}{\Delta t} = \nabla^2 p \quad (4)$$

$$\frac{U_e^* - U_w^*}{\Delta x} + \frac{V_n^* - V_s^*}{\Delta y} = \nabla^2 p \quad (5)$$

An equation for v can also be written similar to equation 3 where equation 4 is obtained using the constraint that u^{n+1} is divergence free. Where U^* and V^* can be defined as

$$U_e^* = \frac{u_P^* + u_E^*}{2} \quad (6)$$

$$V_n^* = \frac{v_P^* + v_N^*}{2} \quad (7)$$

Now, u^* the intermediate velocity is corrected after solving the Pressure Poisson equation along with face velocities (Similarly v^* and V^* can be corrected). The correction is shown below.

$$u^{n+1} = u^* - \Delta t (\nabla p)_{cc} \quad (8)$$

$$U^{n+1} = U^* - \Delta t (\nabla p)_{fc} \quad (9)$$

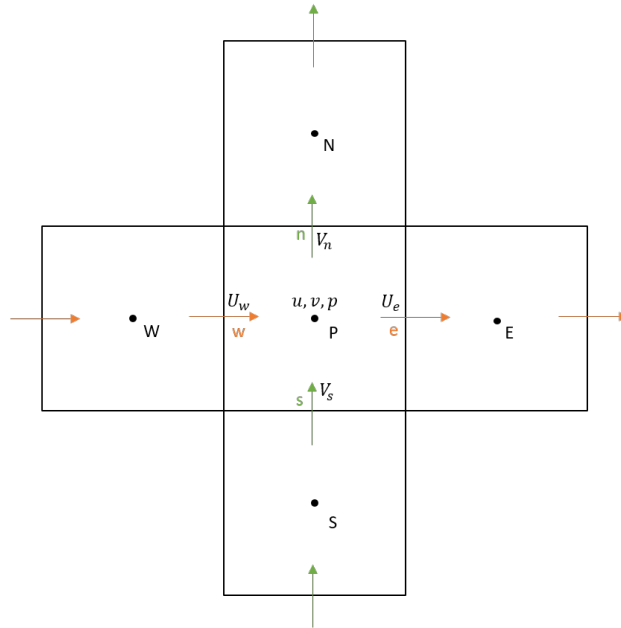


Figure 2: Collocated Grid Arrangement

Mittal et al. [4] observed that an implementation of collocated grid arrangement is easier than the staggered arrangement and a good discrete kinetic energy conservation is achieved when Central Differencing Scheme is used [5]. The system of equations were solved using a Line SoR algorithm for both Advection-Diffusion and Pressure Poisson Equations. The code was written in Fortran 90. Attempt was also made for second order time discretisation using Adam-Bashforth for advection terms and Crank-Nicolson for Diffusion terms. The splitting was done using Fractional-Step method of Van-Kan [6]. So far the results were not satisfactory and improvements are being made. The code can be seen at [GitHub](#)

3 Results and Discussions

The simulations were run until the steady is achieved and then the results obtained were compared and validated against the ones from Ghia et al. First a qualitative comparison is shown using the streamlines for both cases of $Re = 100$ and $Re = 1000$. Fig. 3 shows the streamline for $Re=100$ and subsequently fig. 4a - 4d shows the streamlines obtained for different grid resolutions. It can be seen that lower grid resolutions is not able to resolve the two corner vortices properly.

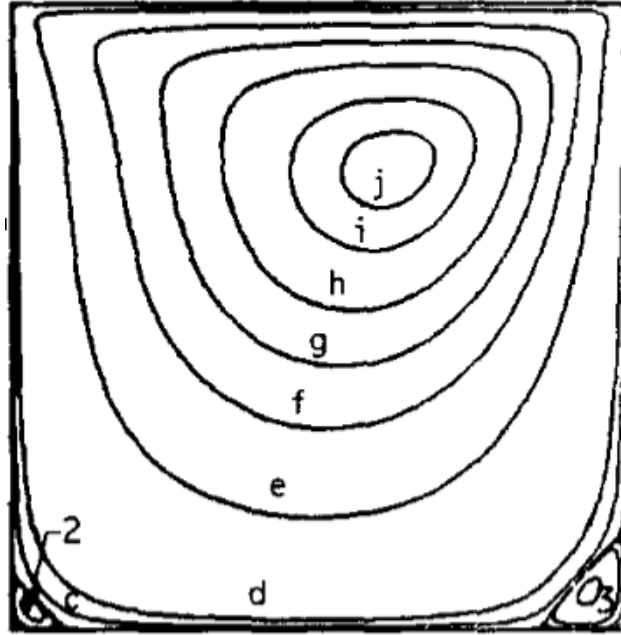


Figure 3: Streamlines by Ghia et al.

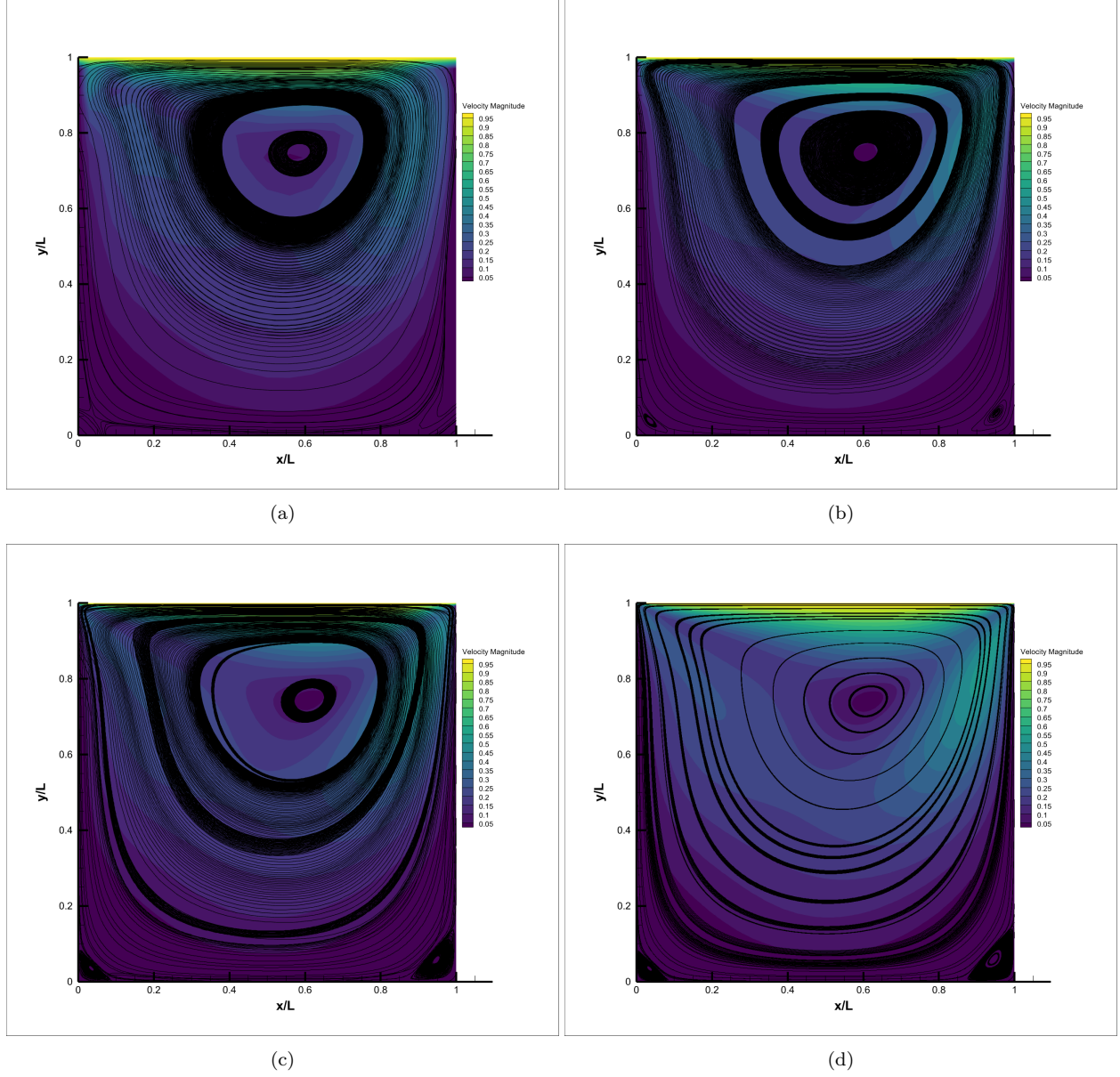


Figure 4: Velocity magnitude contours along with Streamlines at $Re = 100$ for a. 16×16 Grid Points, b. 32×32 Grid Points, c. 64×64 Grid Points, d. 128×128 Grid Points

A qualitative comparison is not enough to show the accuracy of the numerical solution. Therefore, presented next is the variation of v/U along a horizontal line through the geometric center of the cavity (fig. 5) and the variation of u/U along a vertical line through the geometric center of the cavity (fig. 6). It can be seen that the results start to converge as resolution is increased from 16^2 grid points to 128^2 grid points and the data at higher resolution closely matches the validation dataset.

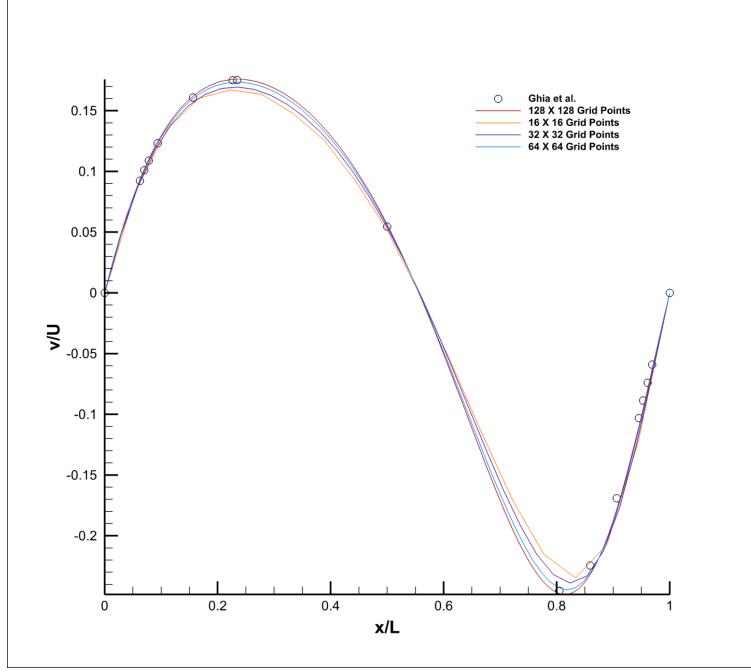


Figure 5: Variation of v/U along a horizontal line through the geometric center of the cavity

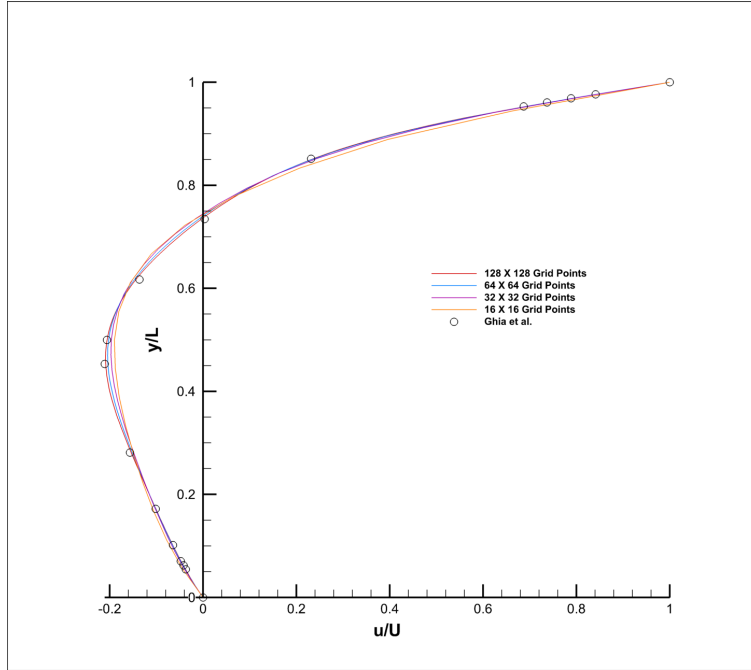


Figure 6: Variation of u/U along a vertical line through the geometric center of the cavity

Presented next is the qualitative comparison of streamlines at $Re=1000$ for different grid resolutions.

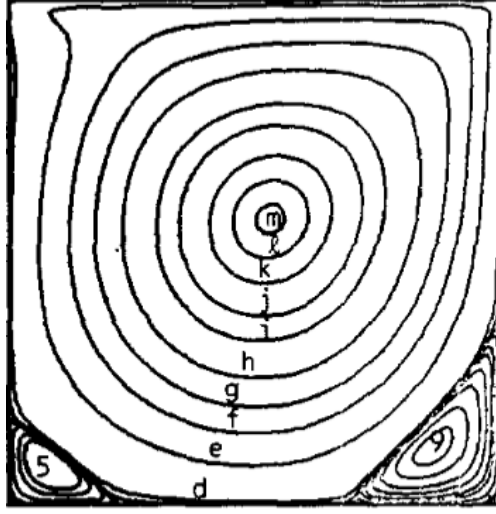


Figure 7: Streamlines by Ghia et al.

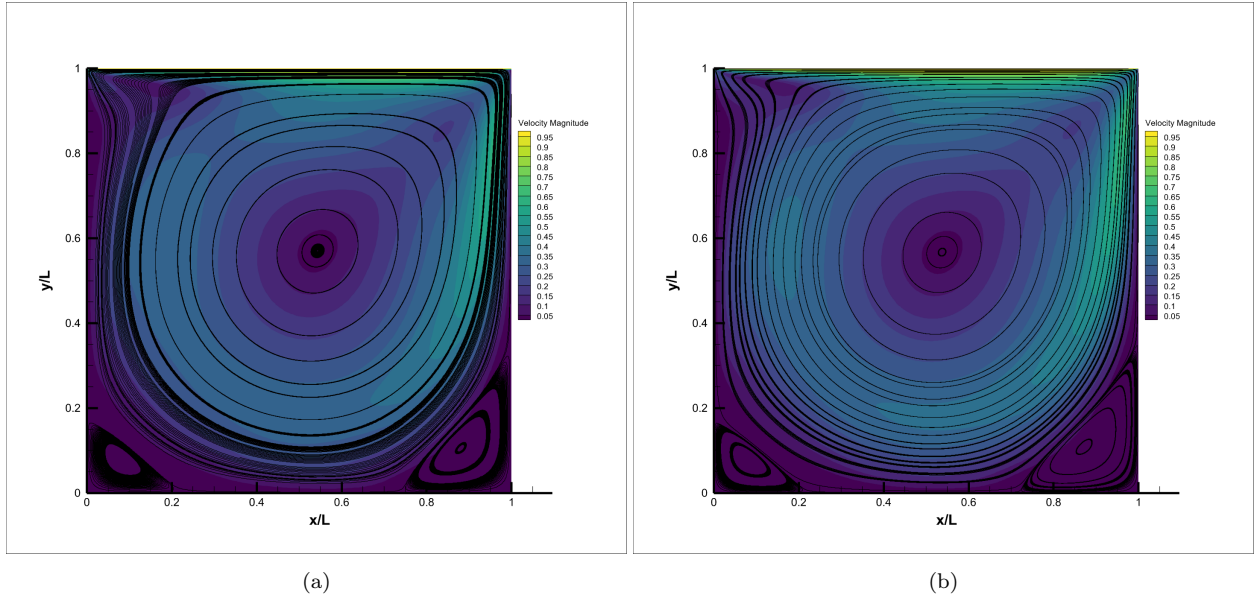


Figure 8: Velocity magnitude contours along with Streamlines at $Re = 1000$ for a. 128×128 Grid Points, b. 256×256 Grid Points

Below shown are the variation of v/U along a horizontal line through the geometric center of the cavity (fig. 9) and the variation of u/U along a vertical line through the geometric center of the cavity (fig. 10) at $Re=1000$. Clearly grid resolution of 128^2 is not enough to match the validation data and resolution has to be increased. Resolution of 256^2 gave a better fit than 128^2 . The data from 512^2 is the most accurate.

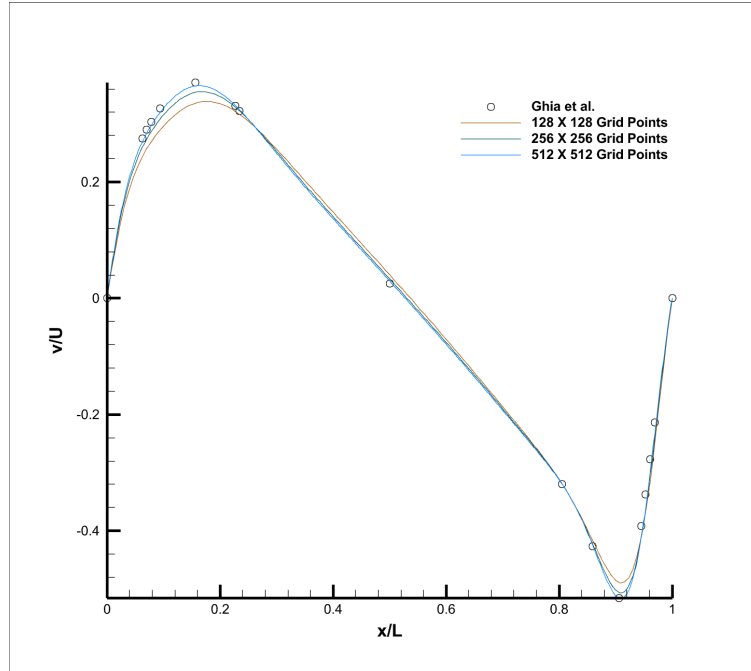


Figure 9: Variation of v along a horizontal line through the geometric center of the cavity

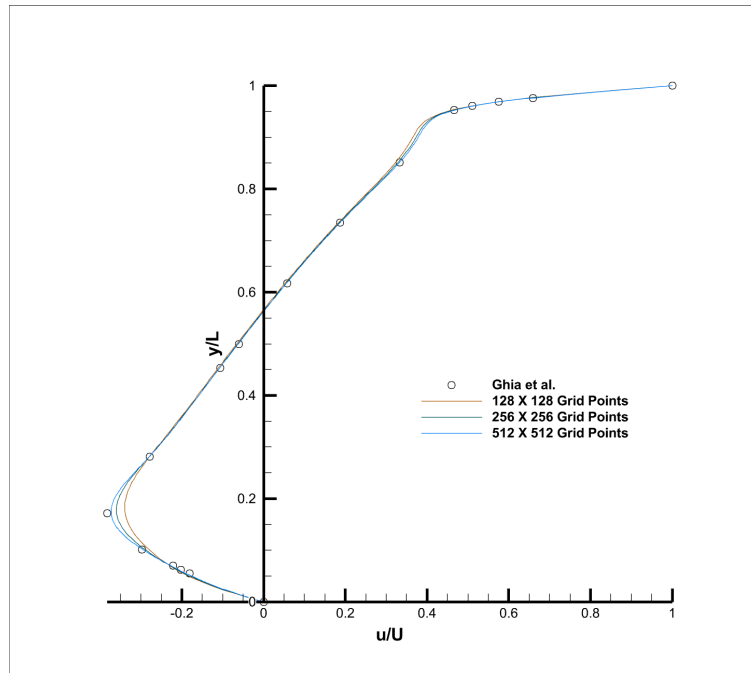


Figure 10: Variation of u along a vertical line through the geometric center of the cavity

References

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