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CONTENTS

STAFF IN-CHARGE

H.O.D.

PRINCIPAL

Introduction

→ Elements of a Communication System

Communication means reading, sending, receiving and processing of information between two or more devices. A collection of elements (devices) which work together to establish communication b/w the sender and receiver is called a communication system.

Some examples of communication system includes radio broadcasting, television broadcasting, radio, telephony, mobile communication, computer communication etc. Two or more people communicating with each other by using sound signal is also known as the communication system.

→ Block diagram of Communication System

The basic components of a communication system are information source, input transducer, transmitter, communication channel, receiver, output transducer and destination.

Information Source

As we know that the communication system establishes the communication bridge b/w the sender and receiver. The communication bridge b/w the sender and receiver to establish this communication bridge b/w the sender and receiver. First we need information to send. This information originates in information source.

The information generated by the source may be in the form of sound (human speech), picture (image source) and words (plain text in particular language). For example if you are talking with your friend on a phone, you are considered as the information source that generates information in the form of sound for beginners to communicate. It's important to understand difference b/w message and information. The message is part of communication which involves

sending information from source to destination. Information is a meaningful data that receiver consumes

Input Transducer

If you want to talk (communicate) with your friend who is sitting beside you, then you can directly talk with him by using voice signal (sound signals). But if the same friend is far away from you, then you can't directly communicate with him by using voice signals (sound signals) because sound signals cannot travel large distance.

So in order to overcome this problem and transmit information to large distance, first we need to convert this sound signal another form of signal (electrical signal or light signal) which travel larger distance. This device is used to convert this sound signal into another (electric) form of signal is called transducer. A transducer is a device which converts from one form of energy or signal into another form of energy or signal. The transducer is present at input side and output side of communication system. Transducer

Generally input transducer converts non-electric signals (physical forms) into an electrical signal.

The best example of an input transducer is microphone which placed b/w the information source & transmitter section. A microphone is a device which converts your voice signal to electrical signals.

Transmitter

It is a device which converts signal produced by source into a form that is suitable for transmission over a given channel or medium. Transmitter uses technique called modulation to convert the electrical signal into a form that is suitable for transmission over a given channel or medium. Modulation is main function of a

$$L = \frac{d}{4} = \frac{c}{4f} = \frac{3 \times 10^8}{4 \times 5 \times 10^3} = 5 \text{ km}$$

It is practical to construct & install an antenna of such height. However this height of antenna may be reduced by modulation techniques & get effective radiation & reception are achieved.

$$L = \frac{d}{4} = \frac{c}{4f} = \frac{3 \times 10^8}{4 \times 3 \times 10^6} = 25 \text{ meters}$$

To remove interference

As an example in Amplitude modulation radio broadcast the max modulating signal frequency permitted is 5 kHz. Amplitude modulation requires bandwidth of 10 kHz for each station or channel. Therefore broadcast channels can be placed adjacent to each other, each channel occupying band width 10 kHz, hence different stations may be allocated bandwidth say from 790 to 800 kHz 800 to 810 kHz and so on in a radio section a tuned ckt i/p select desired station & reject all others.

Increase the signal strength

The baseband signals transmitted by sender are not capable of direct transmission, the strength of message signal should be increased so that it can travel longer distance this is where modulation is essential the most vital need of modulation is to enhance strength of signal without affecting the parameters of carrier signal.

Output transducer

Transducer present at O/P side of system is called O/P transducer. Generally these convert electrical signals into a non electric signal or best example for O/P transducer is loud speaker this converts Electrical signals to voice

Destination

This is final stage of Communication Generally humans at same place are considered as destination for example if you are watching TV, you are considered as destination

Need for Modulation Benefits of Modulation

The message signal or baseband signal is used to modulate a high frequency carrier signal inside the transmitter after modulation the resulting modulated signal is transmitted with help of an antenna which is connected to the O/P side of transmitter. This modulated signal then travel down channel to reach I/p of receiver now one question can why do we use modulation in communication system what will happen if we transmit a message signal or audio signal without modulation. The answer is (that) the modulation serves purpose in a communication as discussed below

Practicality of Antenna

We know that in case of free space it is used as medicine are transmitted & received with help of antennas. messages are transmitted or efficient radiation & reception. The transmitting & receiving antenna must have length comparable to quarter wavelength of frequency used for eg - in AM broadcast system, the maximum audio frequency transmitted from a station is of order 5 kHz if this message audio signal were to be transmitted without modulation

transmitter

when we send the signal to larger distance it undergoes various circumstances which makes signal weak in order to send signal to larger distance without any loss of information it has to undergo process named modulation increases the strength of signal without changing parameter of signal

Communication Channel

The communication channel is a medium through which the signal travels. communication channels are divided into two categories wired & wireless some examples of wired channels includes coaxial cables fibre Optic cables and pair telephones lines. Example of wireless channels are water and vacuum Although channel reduces signal strength. This loss of signal in channel is mainly caused due to external noise, physical surroundings and travel distance. Thus signal received by receiver is very weak. To compensate this signal loss amplifiers are used at both transmitter and receiver side

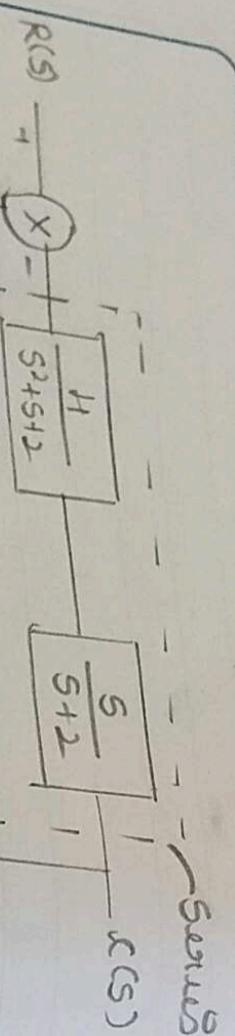
Noise

Noise is unwanted signal that enters the communication system via the communication channel and interferes with transmitted signal the noise signal degrades the transmitted signal

Receiver

The receiver is the device that receives the signal from channel and converts the signal back to its original form which is understandable by humans at destination TV set is example of receiver

Experiment - 1



As the blocks are in series

$$\frac{HS}{(S^2+5+2)(S+2)}$$

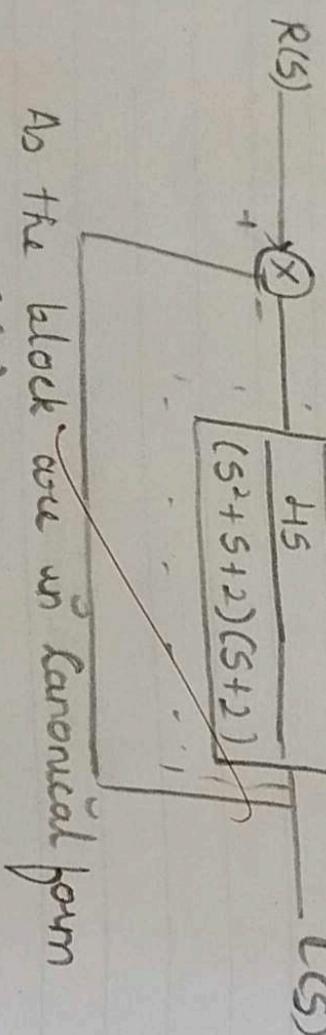
Canonical

$$A = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

$$B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$C = \text{Series}(A, B)$$

$$D = \text{feedback}(C, 1, -1)$$



Result & Conclusion :- The program for the block diagram reduction technique to obtain transfer func' of a control system is successfully executed & hence verified

As the block are in canonical form

$$\frac{C(S)}{R(S)} = \frac{q(S)}{1 + q(S)H(S)}$$

$$= \frac{HS}{(S^2+5+2)(S+2)}$$

$$= \frac{HS}{1 + \frac{HS}{(S^2+5+2)(S+2)}}$$

$$= \frac{HS}{(S^2+5+2)(S+2)+HS}$$

ation

10

6

Li.

tf C

4sc

J=t

cap

Li-

W

Experiment - 2

$$\begin{aligned}
 P &= 4 \\
 Q &= 2 \\
 A &= 4 \\
 R &= 2 \\
 S &= 2 \\
 \end{aligned}$$

Continuous-time transfer function.
Initial condition.

$$\begin{aligned}
 P &= 4 \\
 Q &= 2 \\
 A &= 4 \\
 R &= 2 \\
 S &= 2 \\
 \end{aligned}$$

Continuous-time transfer function.
Initial condition.

$$\begin{aligned}
 P &= 4 \\
 Q &= 2 \\
 A &= 4 \\
 R &= 2 \\
 S &= 2 \\
 \end{aligned}$$

Continuous-time transfer function.
Initial condition.

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Program 2
Implement Signal Flow graph to Obtain transfer function a control system

close all;

clear

$P = H$

$Q = [1 \ 1 \ 2]$

$A = tf(P, Q)$

$R = [1 \ 0]$

$S = [1 \ 2]$

$B = tf(R, S)$

$L = Series(A, B)$

$Av = feedback(L, 1, -1)$

$$\text{no of forward path} = 1 \Rightarrow P_1 = \frac{H}{S^2 + S + 2} \times \frac{S}{S+2}$$

Individual loop gain $\Rightarrow L_1 = \frac{HS}{(S^2 + S + 2)(S+2)}$

Determine of graph $\Delta = 1 - \sum \text{Individual loop gains}$

$$\Delta = 1 + \frac{HS}{(S^2 + S + 2)(S+2)}$$

Theory :- Signal flow graphs usually represent systems components as node & their interaction as branches with gains

Result & Conclusion :- The program for implementation of signal flow graph to obtain transfer function has been successfully done

$= 1$

Finding ΔR

$$\Delta^1 = 1 - [L_1]$$

$= 1 - [0]$

$\therefore TS$ by Mason's formula

$$\frac{L}{R} = \frac{\sum_{k=1}^{\infty} R_k \Delta R}{\Delta} = \frac{P_1 \Delta^1}{\Delta}$$

$$= \frac{HS}{S^3 + 3S^2 + 8S + 4}$$

$$P = 4$$

$$Q = 1 \times 2$$

$$\begin{matrix} 1 & 1 & 2 \end{matrix}$$

A =

$$\begin{matrix} 4 \\ \cdots \\ s^2 + s + 2 \end{matrix}$$

Continuous-time transfer function.

Model Properties

$$R = 1 \times 2$$

$$\begin{matrix} 1 & 0 \end{matrix}$$

$$S = 1 \times 2$$

$$\begin{matrix} 1 & 2 \end{matrix}$$

B =

$$\begin{matrix} s \\ \cdots \\ s + 2 \end{matrix}$$

Continuous-time transfer function.

Model Properties

C =

$$\begin{matrix} 4 & 5 \\ \cdots \\ s^3 + 3s^2 + 4s + 4 \end{matrix}$$

Continuous-time transfer function.

Model Properties

Ans =

$$\begin{matrix} 4 & 5 \\ \cdots \\ s^3 + 3s^2 + 8s + 4 \end{matrix}$$

Continuous-time transfer function.

Model Properties

Experiment - 3

system
at
third step

for D
simulation of poles and zeros of a transfer function

$$\text{num/den} =$$

$$6s + 1$$

$$s + 5$$

$$\text{num} = [6, 1];$$

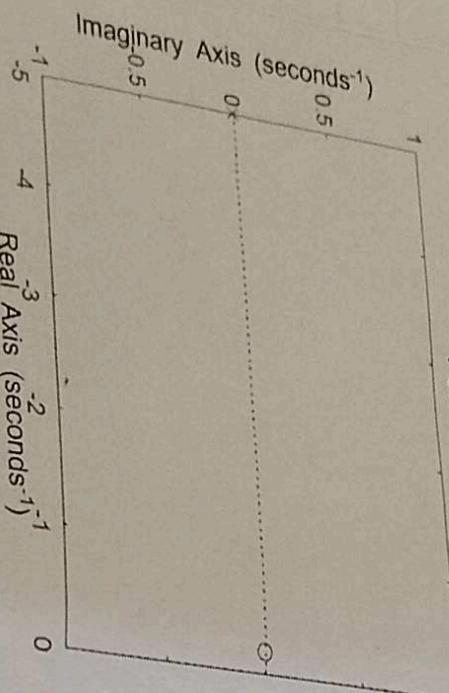
$$\text{den} = [1, 5];$$

$$\text{sys} = \text{tf}(\text{num}, \text{den});$$

~~contour~~(num, den);

[z, p] = ~~tfzp~~(num, den)

pzmap(p, z)



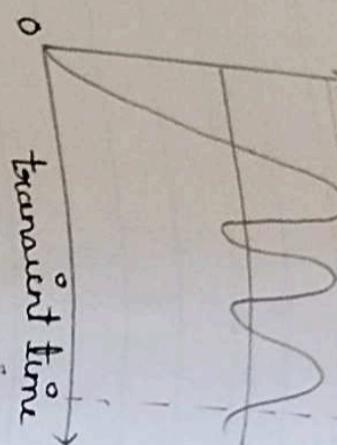
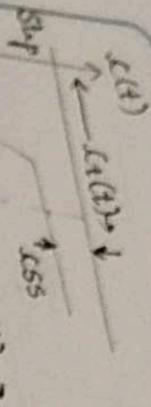
Result & Conclusion :- The program for simulation of poles
zero of a transfer function has been successfully verified

Program -

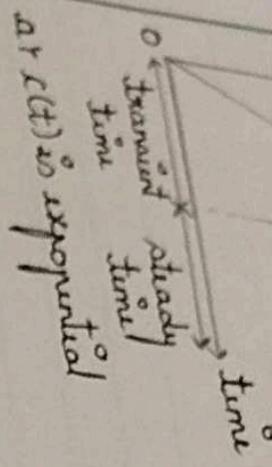
Program - #
is a program
written in C.
It takes two
numbers as input
and prints their
sum.

Program - #
is a program
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Steady state Response $r(t)$



by $L(t)$ is oscillating



Transient Response

peak over time

tolerance band T
in steady state

100%
98%
96%

The total time response $L(t)$ written as

$$L(t) = L_{ss} + L_t(t)$$

Transient Response Specification

4 Delay time (t_d): It is the time required for response to reach 50% of the final value the first attempt

4 Rise time (T_r): It is the time required for the response to rise from 10% to 90% of final value for underdamped systems

4 Peak time (t_p): It is the required for the response to reach its peak value

4 Peak overshoot (M_p): It is the largest error below sequence in O/P during transient period

4 Settling Time (t_s): This is defined as the time required for the response to decrease & stay with specified percentage of its final value

Program 4

1. To determine a time response specification of a second order system for different damping factor under damped system for different damping factor

Software : - MATLAB 2023

Theory : - time response given by the system which is function of the time of the applied excitation called like response of control system

transient response : - The O/P variation during the time taken to achieve the final value called transient response. the time required to achieve the final value called transient response which is called steady state response : - It is that part of the time response which remains after complete transient response vanished from the system O/P

The total time response $L(t)$ written as

$$L(t) = L_{ss} + L_t(t)$$

50%

10%

10%

10%

10%

10%

10%

10%

10%

10%

10%

10%

10%

10%

Equation

$$\omega_n = 10 \text{ rad/s}$$

$$\zeta = 0.5$$

$$t_d = \frac{1+0.7\zeta}{\omega_n}$$

$$tr = \frac{\pi - \theta}{\omega_n}$$

$$tr = \pi - \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta}$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2}$$

$$M_p = e^{-\zeta \pi / \sqrt{1-\zeta^2}} \times 100$$

$$t_p = \frac{T}{\omega_d}$$

$$t_p = \frac{H}{\zeta \omega_n}$$

Calculation

$$\therefore \omega_n = 10 \quad \zeta = 0.5$$

$$t_d = \frac{1+0.7\zeta}{\omega_n} = \frac{1+0.7(0.5)}{10} = 0.135$$

$$\omega_d = \omega_n \sqrt{1-\zeta^2} = 10 \sqrt{1-0.5^2} = 8.66 \text{ rad/s}$$

$$t_p = \pi - \tan^{-1} \left(\sqrt{1-\zeta^2} / \zeta \right) = \frac{\pi - 60}{8.66} = 0.2417 \text{ sec}$$

$$t_p = \frac{\pi}{\zeta \omega_n} = \frac{\pi}{10(0.5)} = 0.8 \text{ sec}$$

$$t_p = \frac{\pi}{\zeta \omega_n} = \frac{\pi}{10(0.5)} = 0.8 \text{ sec}$$

$$\therefore M_p = e^{-\zeta \pi / \sqrt{1-\zeta^2}} \times 100 = e^{-0.5 \pi / \sqrt{1-0.5^2}} \times 100 = 0.1613$$

$$\% M_p = 16.31\%$$

$$M_p = \text{percent overshoot} = 16.30\%$$

Program

$$\omega_n = 10;$$

$$\zeta = 0.5;$$

$$wd = \omega_n * \sqrt{1 - \zeta^2};$$

$$td = (1 + (0.7 * \zeta)) / \omega_n;$$

$$tr = (-\pi - \tan(\sqrt{1 - \zeta^2} / \zeta)) / \omega_d;$$

$$tp = \pi / \omega_d;$$

$$to = 4 / (\pi / \omega_n);$$

$$Mp = exp(-\zeta * pi / \sqrt{1 - \zeta^2}) * 100;$$

cout < "Damping ratio = %." / 10.0 %, zeta);

cout < "damped natural frequency = %." / 10.0 % rad/s, wd);

cout < "delay time = %." / 10.0 % sec, td);

cout < "Peak time = %." / 10.0 % sec, tp);

cout < "Rise time = %." / 10.0 % sec, tr);

cout < "Settling time = %." / 10.0 % sec, ts);

cout < "percent overshoot = %." / 10.0 %, Mp);

$$\mu_n = 5 \quad \xi = 0.7 \quad = \frac{1 + 0.7(0.7)}{5} = 0.298$$

$$td = \frac{1 + 0.7\xi}{\mu_n} = 3.57 \text{ rad/s}$$

$$wd = \mu_n \sqrt{1 - \xi^2} = 5 \sqrt{1 - (0.7)^2} = 3.570$$

$$tr = \pi - \tan^{-1} \frac{\sqrt{1 - \xi^2}}{\xi} = \frac{\pi - 4.536}{3.570}$$

$$tp = \frac{\pi}{wd} = 0.880$$

$$to = \frac{\pi}{\xi wd} = \frac{\pi}{0.7(5)} = 1.142 \text{ sec}$$

$$td = \frac{\pi}{\xi wd} = 1 - 0.7\pi/\sqrt{1 - \xi^2} = 0.046 \times 100$$

$$\begin{aligned} M_p &= 1 - 5\pi/\sqrt{1 - \xi^2} \\ M_p &= 1.6\% \end{aligned}$$

Tabular Column

	$\mu_n = 10 \quad \xi = 0.5$	$\mu_n = 5, \xi = 0.7$
td	0.1350	0.2988
wd	3.57 rad/s	8.66 rad/s
tr	0.3417s	0.6605s (not practical)
tp	0.8808	0.3625s (not practical)
to	1.1425	0.8051s (not practical)
M_p	1.6%	16.31%

Result & Conclusion

The program for determination of time response specification of second order undamped system is written. The theoretical values & practical value are matching with each other

(Q5)

Procedure

- Double click on MATLAB 2023 app to open it
- Take a new script & save the project
- write program for determination of time response
- Run the program & observe the O/P
- Note down all the values
- Calculate all the specification manually & note it down
- check whether the theoretical values & practical values are matching

Method for
Damping Ratio
and Natural Frequency

$$\text{Damping Ratio} = 0.1$$

$$\text{Damped Natural Frequency} = 9.95 \text{ rad/s}$$

$$\text{Rise Time} = 0.32 \text{ s}$$

$$\text{Settling Time} = 4.02 \text{ s}$$

$$\text{Percent Overshoot} = 72.92 \%$$

$$\text{Damping Ratio} = 0.3$$

$$\text{Damped Natural Frequency} = 9.54 \text{ rad/s}$$

$$\text{Rise Time} = 0.35 \text{ s}$$

$$\text{Settling Time} = 1.40 \text{ s}$$

$$\text{Percent Overshoot} = 37.23 \%$$

$$\text{Damping Ratio} = 0.5$$

$$\text{Damped Natural Frequency} = 8.66 \text{ rad/s}$$

$$\text{Rise Time} = 0.42 \text{ s}$$

$$\text{Settling Time} = 0.92 \text{ s}$$

$$\text{Percent Overshoot} = 16.30 \%$$

$$\text{Damping Ratio} = 0.7$$

$$\text{Damped Natural Frequency} = 7.14 \text{ rad/s}$$

$$\text{Rise Time} = 0.62 \text{ s}$$

$$\text{Settling Time} = 0.80 \text{ s}$$

$$\text{Percent Overshoot} = 4.60 \%$$

$$\text{Damping Ratio} = 0.9$$

$$\text{Damped Natural Frequency} = 4.36 \text{ rad/s}$$

$$\text{Rise Time} = 1.65 \text{ s}$$

$$\text{Settling Time} = 1.02 \text{ s}$$

$$\text{Percent Overshoot} = 0.15 \%$$

Experiment - 5

$$\text{Given } \omega_n = 10 \text{ rad/sec}$$

$$\xi = 0.5$$

$$H(s) = \frac{\omega_n^2}{R(s)} = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$$H(s) = \frac{10^2}{s^2 + 2(0.5)(10)s + 10^2}$$

$$H(s) = \frac{100}{s^2 + 10s + 100} \quad |s \rightarrow j\omega$$

$H(j\omega)$

$$= \frac{100}{j\omega)^2 + 10(j\omega) + 100}$$

$$= \frac{100}{-\omega^2 + j10\omega + 100}$$

$$= \frac{100}{(100 - \omega^2) + j10\omega}$$

magnitude

$$|H(j\omega)| = \sqrt{(100 - \omega^2)^2 + j10\omega}$$

$$H(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2} \quad |s \rightarrow j\omega$$

$$H(j\omega) = \frac{100}{\sqrt{(100 - \omega^2)^2 + 100\omega^2}}$$

phase plot

The phase of any complex conjugate is found with old formula

$$\tan^{-1}(\frac{b}{a})$$

where b is imaginary part

a is real part

\therefore The phase is found as

$$H(j\omega) = \tan^{-1}(\frac{10\omega}{100 - \omega^2})$$

$$LH(j\omega) = \tan^{-1}\left(\frac{10\omega}{100 - \omega^2}\right)$$

$$LH(j\omega) = \tan^{-1}\left(\frac{10\omega}{100 - \omega^2}\right)$$

Program - 5
Aim : Determine the frequency response of a second order system

software used : MATLAB 2023

Theory : The second order systems are the systems or network which contain two or more storage elements and have describing equations that are second order differential equations so

The second order system is given by a transfer function

$$H(s) = \frac{\omega_n^2}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

Ex :- Determine the frequency response of a 2nd order system given the natural frequency of the system is 10 rad/sec & damping ratio $\xi = 0.5$

Tabular Column for magnitude Plot

ω	$20 \log_{10} H(j\omega) / \text{dB}$
0	0
0.1	0
0.2	0
0.3	-11.3
0.4	-18.63
0.5	-23.83
1	-27.78
2	-39.89
3	-50.04
4	-59.19
5	-64.08
10	-67.95
20	-80
50	-100
100	-120
200	-130
300	-134
400	-135
500	-136
1000	-137

Program

```

w = 10;
zeta = 0.5;
num = wn^2;
den = [1 2 * zeta * wn wn^2];
sys = tf([num, den]);
w = logspace(-1, 2, 1000);
[mag, phab] = bode(sys, w); → To give data of magnitude & phase
magdb = xmag(mag, 1), sys(mag, 5); → 5 values on plot
phase = xphase(phab[1], size(phab, 3));
figure;
subplot(2, 1, 1); → To conv. magnitude to decibel
semilogx(w, 20 * log10(magdb)); → To conv. phase to radian
grid on;
title('magnitude response');
xlabel('frequency (rad/s)');
ylabel('magnitude (dB)');
subplot(2, 1, 2); → Second position
semilogx(w, phase);
grid on;
title;
xlabel('Phase response');
ylabel('frequency (rad/s)');

```

Phase

$$|H(j\omega)| = -\tan^{-1} \left(\frac{10\omega}{100-\omega^2} \right)$$

Tabular Column for phase plot

ω

0

0.1

0.2

0.3

0.4

0.5

1

2

3

4

5

6

7

8

9

10

11

12

13

14

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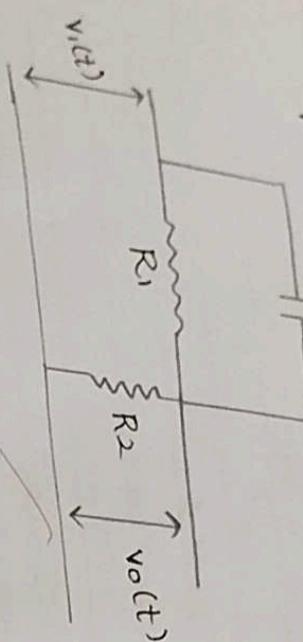
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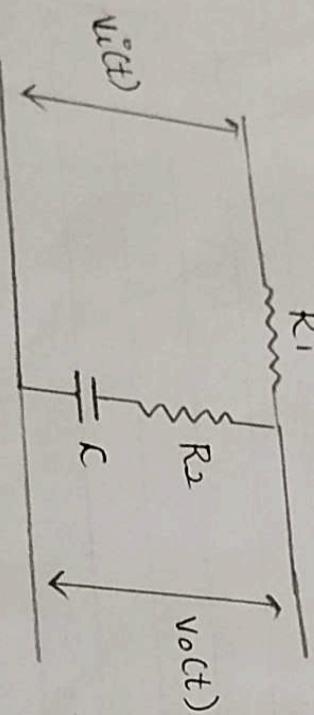
197

Circuit diagram

1) Lead Compensator



2) Lag Compensation



Program - 6
Aim: Determination of frequency response of a lead lag compensated system

Software used : MATLAB

Theory

A lead lag compensation is a component in control system that improves an undesirable frequency response in a feedback control system. Both lead compensation and lag compensation introduce a pole-zero pair into the open loop transfer function in Laplace transform as $\frac{Y}{X} = S - p$

where

$X \rightarrow s/p$ to the compensation

$Y \rightarrow o/p$

$S \rightarrow$ complex Laplace transform

$\tau \rightarrow$ the zero frequency

$p \rightarrow$ the pole frequency

The pole & zero are both typically negative are left side of origin in the complex plane. In a lead compensation $|z| < |p|$ while in a lag compensation

the overall T.F is written as $\frac{Y}{X} = (S - z_1)(S - z_2)$

Typically $|p_1| > |z_1| > |z_2| > |p_2|$ where z_1, z_2, p_1, p_2 are the zeros and poles of lead compensator of τ_1, τ_2 are the zeros and poles of lag compensator

Program

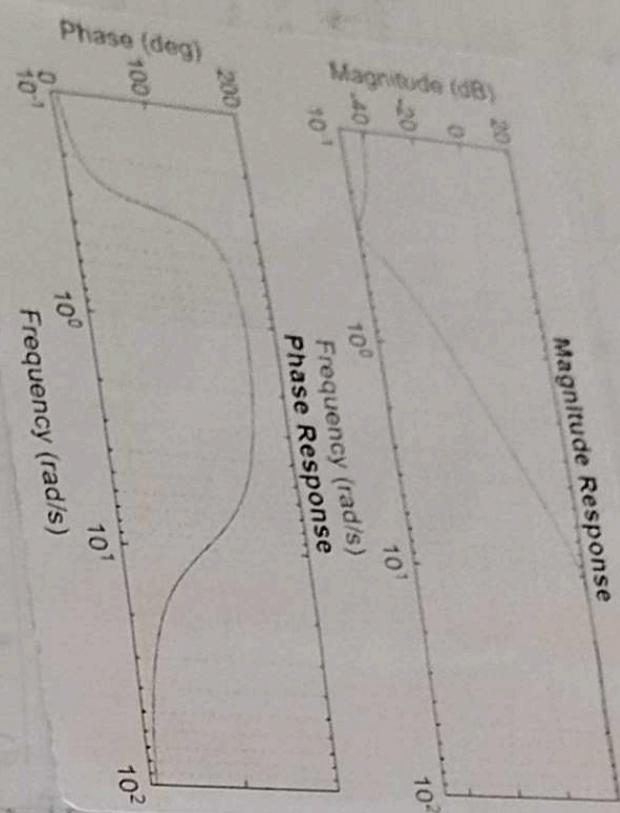
```

k=1;
wn = 10;
m = 10;
zeta = 0.5;
alpha = 0.1;
beta = 10;
num = m * alpha * beta * k * m * alpha * k; [alpha*beta k*m*alpha]
den = [1 2 * zeta * wn, wn^2];
sys = tf([num, den]);
w = logspace(-1, 2, 1000);
[mag, phase] = bode(sys, w);
mag2d = abs(mag);
phase2d = rad2deg(phase);
figure
subplot(2, 1, 1);
semilog (w, 20 * log10(mag2d));
grid on;
title ("Magnitude Response");
xlabel ('Freq (rad/sec)');
ylabel ('Magnitude (dB)');
subplot (2, 1, 2);
semilog (w, phase2d);
grid on;

```

Result & Conclusion

The program for determination of frequency responses of a lead lag compensator is written and executed in MATLAB graph are obtained as shown



Characteristic polynomial:

$$-s^4 + 8s^3 - 18s^2 + 16s + 5$$

South array -
I 18
II 0

144 00
2304 00
a 00

Routh array = 5×3

144
2304
θ 0 0 0

-Routh array = $\begin{matrix} & & & 18 \\ & & 1 & 5 \\ 8 & & & \\ 16 & & & \\ & 0 & & \end{matrix}$

144
0
0

Routh Array:

1.00	18.00	5.00	8.00	16.00
------	-------	------	------	-------

0.00e+00
144.00 40.00

304.00 0.00e+00

1160.00 |
0.00e+00

The system is stable.

Experiment - 4

RH criteria table
 $C_E = S^4 + 8S^3 + 18S^2 + 16S + 5 = 0$

S^4	1	18	5
S^3	8	16	0
S^2	16	5	0
S^1	13.81	0	0
S^0	5	0	0

From RH table it is observed that there is no sign change in 1st column, hence the system is stable.

Hence the system is stable.

Program:
 Aim: Using suitable package, obtain the time response from state model of system

Software used : MATLAB 2023

Program

```
A = [-1, -1; 1, 0];
B = [1; 0];
C = [0, 1];
D = 0;
```

```
sys = ss(A, B, C, D);
```

y: Desired i/p signal

```
t = linspace(0, 10, 1000);
```

```
u = sin(n(t));
```

% Stimulate system response

```
[y, t, x] = lsim(sys, u, t);
```

figure;

```
plot(t, y);
grid on;
```

legend('input', 'output');

Result & Conclusion

Hence simulation package study of speed control of DC motor is proved in form of graph.

$$x = \begin{bmatrix} -1 & -1 \\ 1 & 0 \end{bmatrix}, B = \begin{bmatrix} 1 \\ 0 \end{bmatrix}, C = \begin{bmatrix} 0 & 1 \end{bmatrix}$$

$$\text{Given } x = \begin{bmatrix} -1 & -1 \\ 1 & 0 \end{bmatrix}$$

$$T_F = C(CB - x)^{-1}B + D$$

$$CB - x = \begin{bmatrix} 5 & 0 \\ 0 & 5 \end{bmatrix} - \begin{bmatrix} -1 & -1 \\ 1 & 0 \end{bmatrix}$$

$$(CB - x)^{-1} = \frac{1}{5+5+1} \begin{bmatrix} 5 & 1 \\ -1 & 5 \end{bmatrix} = \begin{bmatrix} 5 & -1 \\ 1 & 5+1 \end{bmatrix}$$

$$T_F = [0 \ 1] \begin{bmatrix} 5 & -1 \\ 1 & 5+1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} + D$$

$$T_F = [0 \ 1] \begin{bmatrix} 5 & -1 \\ 1 & 5+1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} + \frac{1}{5+5+1} \begin{bmatrix} 5 \\ 1 \end{bmatrix}$$

$$T_F = \frac{1}{5+5+1}$$

Method of finding transfer function
with respect to input and output

Ans

Example : Root locus for a negative feedback control system having

1. sketch the root transfer function

Open loop

$$G(s)H(s) = \frac{K}{s(s+5)(s+10)}$$

Step 1: no of poles : $P=5, S=0, S=5, S=-10$

no of zeros : $Z=0$

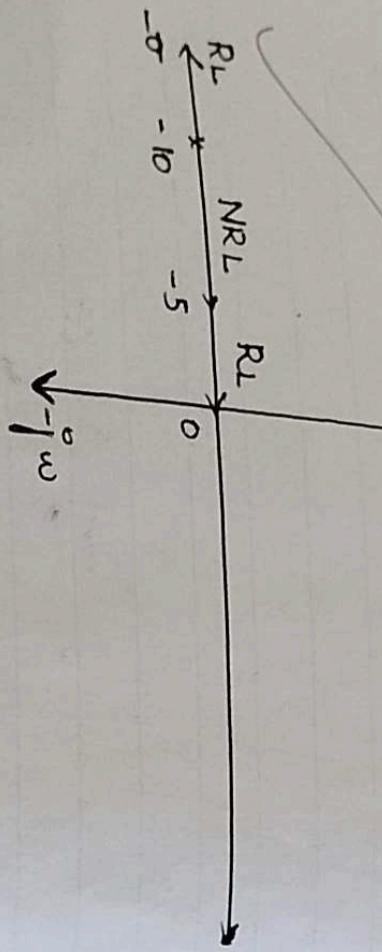
Here $P > Z$ $Z > 0$

Root locus start from open loop poles & terminate at loop zeros

Starting pt of root locus: $0, -5, -10$

Ending pt of root locus: ∞, ∞, ∞

Step 2: Plotting the root locus (RL) regions & non root locus (NRL)



Theory : Root Locus: It starts from poles and at zeros to find coefficient of poles, equal to the sum of 0 & take the coefficient of poles equal to the sum of 0 & take the coefficient in descending order of power of s

$$G(s)H(s) = \frac{K \cdot 1}{s(s+5)(s+10)}$$

$$\text{denominator: } s(s+5)(s+10)=0$$

$$s^3 + 10s^2 + 50s + 50 = 0$$

$$\text{deno co-eff} = [1, 10, 50, 50]$$

$$\text{numerator co-eff} = [1]$$

Block plot: given $G(s)H(s) = 50$

$$\text{denominator} = s(0.5s+1)(0.5s+1)$$

$$(0.5s^2)(0.5s+1) = 0$$

$$0.25s^2 + 0.5s^2 + 0.05s^2 + s$$

$$0.025s^2 + 0.5s^2 + 0.5s + 0s = 0$$

denominator coefficient = [0, 0.25, 0.55, 1, 0]

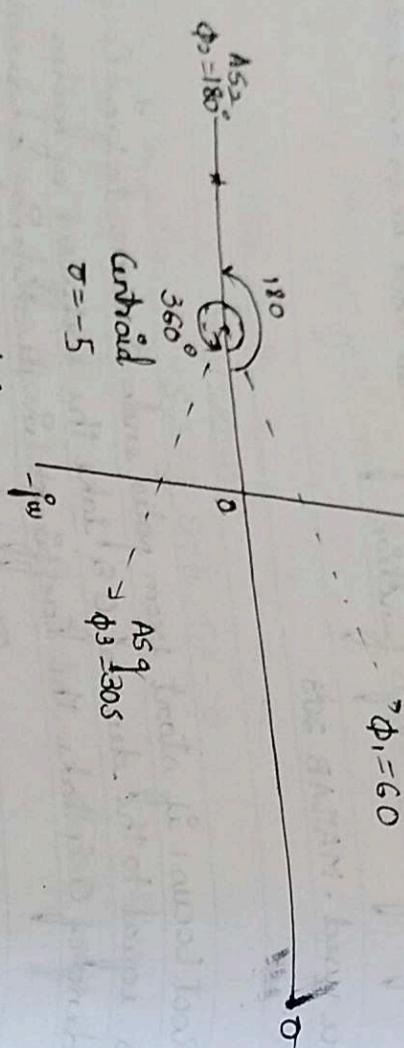
numerator co-eff = [50]

$$\phi_1 = \left(\frac{2 \times 0 + 1}{3}\right) \times 180 = 60^\circ$$

$$\phi_2 = \left(\frac{2 \times 1 + 1}{3}\right) \times 180 = 180$$

$$\phi_3 = \left(\frac{2 \times 2 + 1}{3}\right) \times 180 = 300$$

Program for root locus



Step 4: Centroid of root locus

$$\bar{\sigma} = \frac{\sum \text{real part of poles} - \sum \text{real part of zeros}}{p-z}$$

$$\bar{\sigma} = \frac{(0 - 5 - 10) - (0)}{3-0} = -5$$

Step 5: To find BAP

$$1 + G(s)H(s) = 0$$

$$\frac{1 + K}{s(s+2)(s+10)} = 0$$

$$(s^2 + 5s)(s + 10) + K = 0$$

$$15 = -(s^2 + 15s^2 + 50s)$$

$$\frac{ds}{db} = 3s^2 + 30s + 50$$

$$S_1 = -2.1 \quad S_2 = -2.1$$

$$K/J = -2.1 = 90.76$$

Step 6 Interaction with imaginary axis consider L.C.
 $1 + g(s) H(s) = 0$
 $s^3 + 15s^2 + 50s + k = 0$

$$s^3 + 15s^2 + 50s + k = 0$$

$$s^3 + \frac{15}{5}s^2 + \frac{50}{5}s + \frac{k}{5} = 0$$

$$s^3 + 3s^2 + 10s + \frac{k}{5} = 0$$

$$\frac{k}{5} = -10s - 3s^2 - s^3$$

$$k = -50s - 15s^2 - 5s^3$$

$$k_{\max} = -50$$

frequency of oscillation where root locus intersects the imaginary axis

$$15s^2 + k = 0$$

$$15s^2 = -k$$

$$i\omega$$

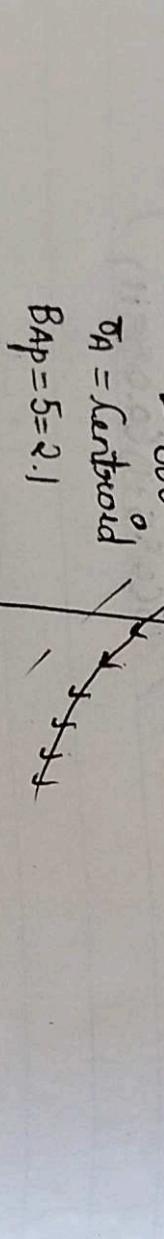
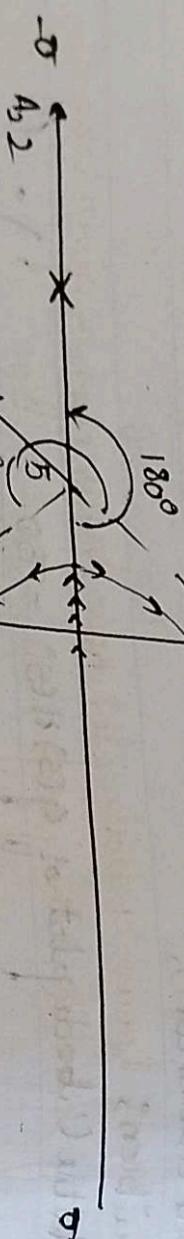
As

Result

From the bode magnitude & phase diagram

$$GM = -8dB$$

$$PM = \phi_M = -15.25^\circ$$



$$B_{Ap} = 5 = 2.1$$

$$A_{S_1}$$

Result
 The root locus & bode plot are drawn for a given transfer function by writing the program

Ex: investigate the stability of a -ve feedback control system where Open loop + + is given by

$$g_{CS}H_{CS} = \frac{50}{50(0.55+D)(0.055+1)}$$

$$\text{put } s=j\omega \quad g(j\omega)H(j\omega) = \frac{50}{j\omega(0.5j\omega+1)(0.5j\omega+1)}$$

$$Y(j\omega) = \frac{50}{j\omega}$$

$$20\log|Y(j\omega)| = 20\log 50 - 20\log\omega = 54 \text{ dB} -$$

~~Tabular Column~~
Factor Corner Frequency Magnitude & slope characteristic of
~~frequency~~ The factor for a magnitude = 54 dB at
 $\omega = 0.1 \text{ rad/sec}$ slope = -20 dB/dec

$$\frac{50}{j\omega}$$

$$\frac{1}{1+0.5j\omega}$$

$$\omega_2 = 2$$

Net slope b/w ω_1 & ω_2 = slope contributed by $(1+0.5j\omega)$ for $\omega > \omega_1$ provided slope =

$$20 = -20 - 20 = -40 \text{ dB/dec}$$

~~During~~

Net slope w2 onwards slope contributed by $(1+0.05j\omega)$ for $\omega > \omega_2$ provides slope = $-20 - 40 = -60 \text{ dB/dec}$

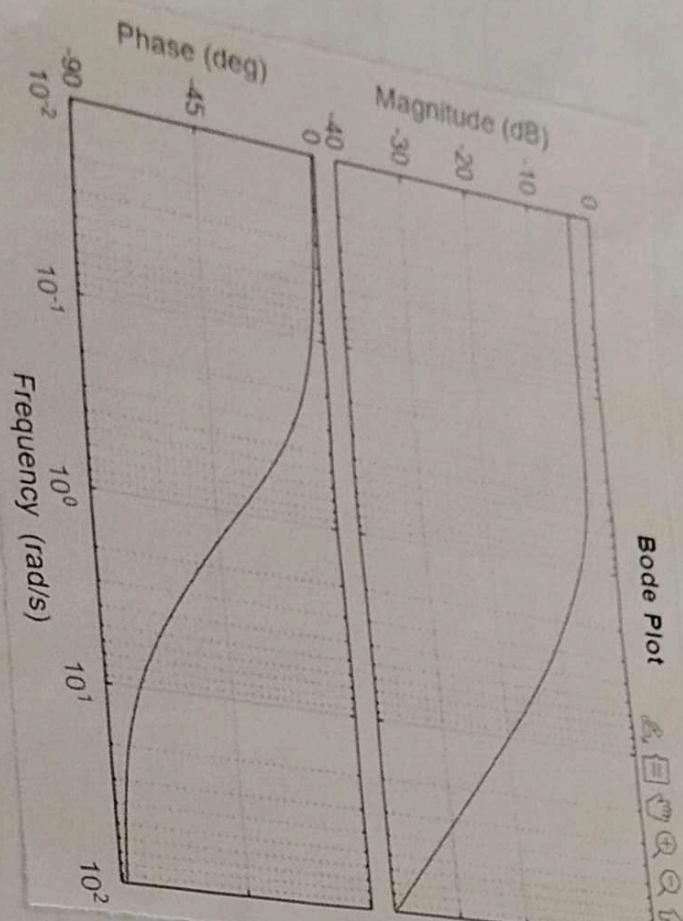
$$\frac{1}{1+0.05j\omega} \quad \omega_2 = \frac{1}{0.005}$$

$$\omega_2 = 20$$

Experiment - 9

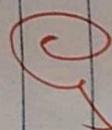
Program - 9
Aim - Analyze the stability of given system bode plot

Theory - Bode plots graphically represent a system frequency response, producing insights into stability by determining if the gain & phase margins are concave a system subjected to disturbance



```
num = [1, 2];
den = [1, 4, 3];
sys = tf(num, den);
figure,
bode(sys);
grid on;
title('Bode Plot');
```

Result & Conclusion :- The program for analysing the stability of given system using pole is already fully executed & verified



Experiment - 90

Experiments done by
K. L. E. Institute of Technology
Date : 10/10/2018
Page No. 26

K.L.E. INSTITUTE OF TECHNOLOGY, HUBBALLI.

Program - 10

Program :- Analyze the stability of given system using nyquist plot

Author :- MATLAB 2024

Software :- MATLAB 2024

Theory :- Nyquist plot maps system's frequency response automatically. Stability is known by analyzing the plots. The measurement by $-1 + j0$ point

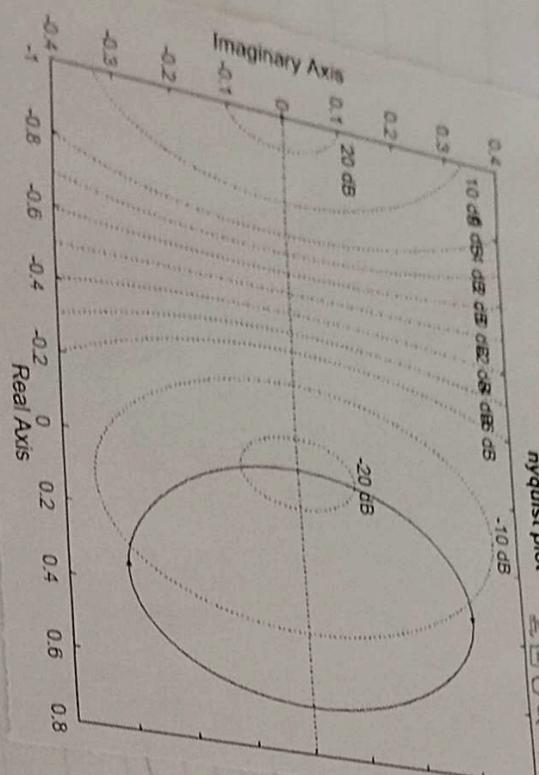
```
num = [1, 2];
```

```
den = [1 4 3];
```

```
sys = tf(num, den);
```

```
figure;
nyquist(sys);
grid on;
```

```
title('Nyquist Plot');
```



Result & Conclusion - The program to analyze the stability of given system using nyquist plot was successfully executed & verified.

Welding
Welding
Welding
Welding
Welding

Experiment - 11

Welding
Welding

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Welding
Welding

$$X = \begin{bmatrix} -1 & -1 \\ 1 & 0 \end{bmatrix} \quad B = \begin{bmatrix} 1 \\ 0 \end{bmatrix} \quad C = \begin{bmatrix} 0 & 1 \end{bmatrix}$$

$$\text{Given } X = \begin{bmatrix} -1 & -1 \\ 1 & 0 \end{bmatrix}$$

$$T_F = C(CS - \alpha)^{-1}B^T P$$

$$SD - \alpha = \begin{bmatrix} S & 0 \\ 0 & S \end{bmatrix} - \begin{bmatrix} -1 & 0 \\ 1 & 0 \end{bmatrix}$$

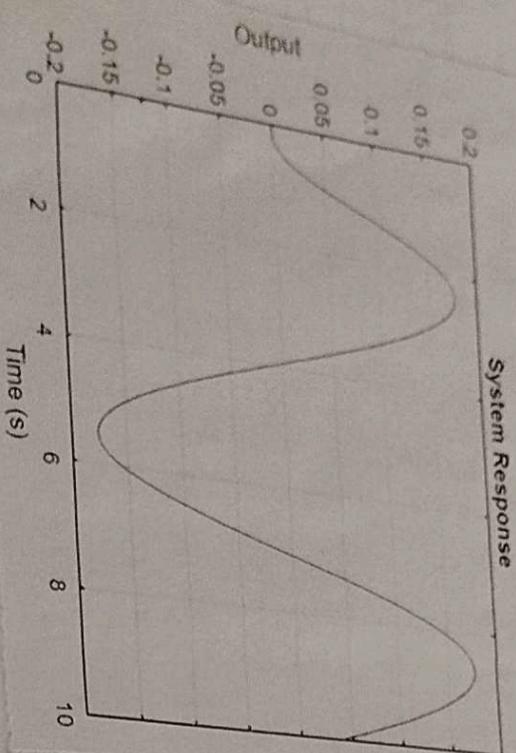
$$= \begin{bmatrix} S+1 & 1 \\ -1 & S \end{bmatrix}$$

$$(SD - \alpha)^{-1} = \frac{1}{S^2 + S + 1} \begin{bmatrix} S & -1 \\ 1 & S+1 \end{bmatrix}$$

$$TP = [0 \ 1] \begin{bmatrix} S & -1 \\ 1 & S+1 \end{bmatrix} \begin{bmatrix} 1 \\ 0 \end{bmatrix} + 0$$

$$= [0 \ 1] \frac{\begin{bmatrix} S \\ 1 \end{bmatrix}}{S^2 + S + 1}$$

$$TF = \frac{1}{S^2 + S + 1}$$



Experiment - 1Q

Given
Initial condition
 $x(0) = 0.18$
Final condition
 $x(7.5) = 0$
Damping ratio
 $\zeta = 0.5$
Natural frequency
 $\omega_n = 1.5 \text{ rad/s}$

Program 12
Implementation of PI, PD Controllers

PI Controller

$\star \text{tf}('s')$,

$\star \text{t}((s+1))$

$G = 1/(s * (s+1))$

$G = 1;$

$K_p = 1;$

$K_d = K_p + K_d^o / s;$

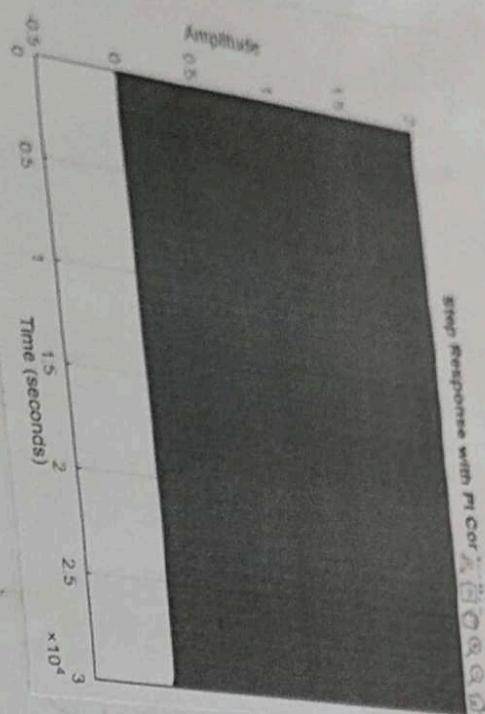
$L = K_p + K_d^o / s;$

$Sy^o = \text{feedback}(G, L);$

$Stf^o = \text{tf}(Sy^o);$

$grid on;$

$title('Step Response with PI Controller');$



* PD Controllers

$S = \text{tf}('s');$

$G = 1/(s * (s+1));$

$G = 1;$

$K_p = 1;$

$K_d = 1;$

$L = K_p + K_d * S;$

$Sy^o = \text{feedback}(G, L);$

$Stf^o = \text{tf}(Sy^o);$

$grid on;$

$title('Step response with PD controller');$

Theory & PC Controller :- Combines proportional (P) and integral (I) terms

* PD Controller :- Combines proportional (P) & derivative (D) terms

PD Controller
 $G = \frac{1}{s + 1}$

$$G = \frac{1}{s + 1};$$

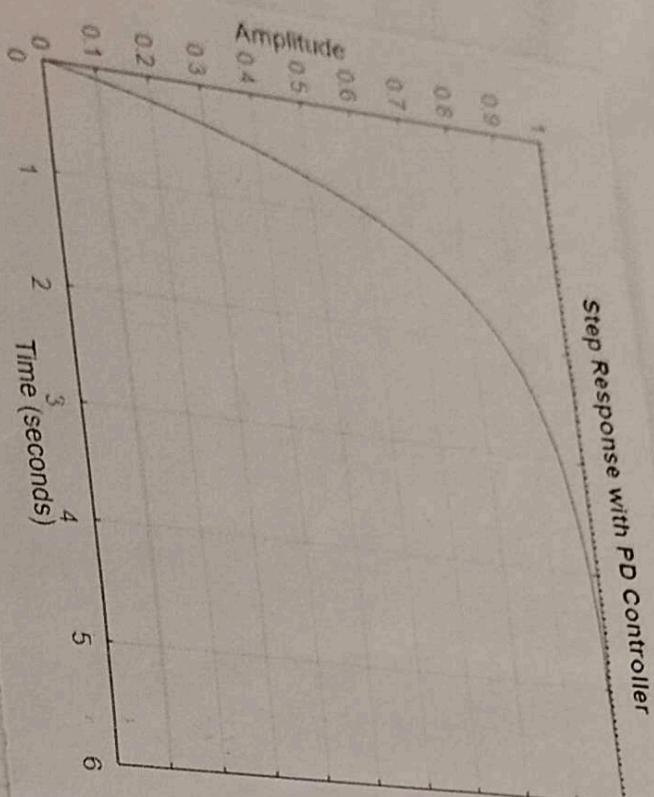
$$k_p = 1;$$

$$k_d = 4;$$

$$C = k_p + k_d * s;$$

$$sys = feedback(C * G, 1);$$

~~grub on~~
 title('Step Response with PD Controller');



Result & Conclusion :- The program to implement PI & PD

controller has been executed successfully verified

