

Study of Capillary Driven Flow in an Interior Corner of Rounded Wall Under Microgravity

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Abstract The capillary driven flow in cylindrical interior corners satisfying the Concus-Finn condition was investigated under microgravity. The governing equation of capillary driven flow in cylindrical interior corners was established, and the approximate analytical solution was obtained. The relationship between liquid's front position and time was derived, which was then compared with the results of drop tower experiments and numerical simulation using the FLOW-3D software. The influence of different parameters on the interior corner flow was studied. The results showed that the meniscus height decreased as contact angle increased, and increased as the radius of rounded wall increased. The influence of decreasing the contact angle on a rounded wall was greater than that on a straight wall. Our findings can be referred to designing tanks or choosing the suitable solution in the space fluid management.

Keywords Rounded wall · Capillary driven flow · Approximate analytical solution · Liquid's front position

List of symbol

Dimensional quantities

x', y', z'	rectangular coordinate system
u', v', w	velocities of x', y', z' -component
H	constant height of meniscus at initial location
L	characteristic length of the fluid column
R'_1	radius of meniscus at z -location
R'_2	radius of rounded wall
μ	dynamic viscosity
ρ	density
σ	surface tension
θ_1	contact angle of straight wall
θ_2	contact angle of rounded wall
α	angle between OO_1 and y -axis
P'	Pressure
t'	Time
h'	the height of meniscus along OO_1 (the minimum distance from point O to meniscus)
k	the slope of OA in the $O\xi\eta$ coordinate frame
Θ	$\varepsilon^2 \sigma \rho H / \mu^2$
$C_{I,i}, C_{II,i}$	the coefficient
d_k	the coefficient of $h(\lambda)$ by series expansion
a_j	the coefficient of A by series expansion
c	a constant value which is stand by $z_f \dot{z}_f$ in Eq. 25
χ	the relative error of area A
δ	the relative error δ of $w_{0II}(\xi, \eta)$ in the rounded wall OB

Dimensionless quantities

x, y, z	rectangular coordinate system, $x = x'/H$, $y = y'/H$, $z = z'/L$
ε	ratio of length scale, $\varepsilon = H/L$

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W	velocity coefficient, $W = \varepsilon\sigma/\mu$
u, v, w	velocities of x, y, z -component, $u = u'/\varepsilon W$, $v = v'/\varepsilon W$, $w = w'/W$
P	pressure, $P = HP'/\sigma$
t	time, $t = Wt'/L$
x_0	the value along the x -axis on the meniscus, as shown in Fig. 4
$S(x, h)$	the value along y -axis at a given x_0 , as shown in Fig. 4
R_1	radius of meniscus at z -location, $R_1 = R'_1/H$, as shown in Fig. 4
R_2	radius of rounded wall, $R_2 = R'_2/H$, as shown in Fig. 4
A	cross-sectional flow area
Q'	volumetric flow rate
x_1, y_1	center coordinates of meniscus, as shown in Fig. 4
x_2, y_2	center coordinates of rounded wall, as shown in Fig. 4
x_A, y_A	coordinate of intersection between meniscus and straight wall, as shown in Fig. 4
x_B, y_B	the coordinate of intersection between meniscus and rounded all, as shown in Fig. 4
h	$h = h'/H$
z_f	liquid's front position at t
z_{f0}	liquid's front position at initial time
λ	$\lambda = z/z_f$
ξ ,	local coordinates, the intersection point of η -axis and
η	meniscus is point C , $OC = h$, as shown in Fig. 4
ξ_0, η_0	coordinate transform of (x_0, S) on local coordinate system
g_0	gravitational acceleration in the locality

Introduction

Capillary driven flow is common and can be observed in our surroundings from the draining of fluids via a sponge or towel to the wicking of candle wax. Although capillary phenomena are often ignored in large-scale systems considering the great Earth's gravity, they are significant in non-gravity environments. There is the potential to displace fluids using only surface tension and wetting properties, reducing the need of intricate systems with moving parts or higher risk of failure such as centrifugal pumps. Hence, capillary driven flow has the advantages of economy and safety.

Spontaneous capillary driven flow can be used to transport fluids in places with the less influence of gravity. The amount and rate of the transported fluid strongly depends on the system geometry. As a matter of fact, due to the

induced pressure gradient caused by the decrease of mean radius of curvature as it approaches the corner, interior corners can also be exploited to pump fluid passively. Research on capillary flow in interior corner can be traced back to the 1960s. Concus and Finn (1969) proposed the critical contact angle of the wetting liquid, i.e. Concus-Finn condition under non-gravity, and classified capillary flow in interior corners as steady and unsteady solutions. Weislogel (1996) simplified the three dimensional Navier-Stokes equations to one dimensional case, and got the solution by introducing the lubrication approximation. For special cases of constant height boundary condition, the length of the fluid column increases in the interior corner ($L \propto t^{1/2}$). After that, Weislogel (2002) and Nardin (2005) published series of papers to extend the theory to complex geometry calculation. Mason and Morrow (1984, 1991) derived an expression for the displacement curvature of the main terminal meniscus in a regular n -sided tube with any contact angle, and later reported the general solutions for completely wetted triangular tubes of different shapes. Dong and Chatzis (1995) took the wetting area as variable to investigate the flow in square capillary tube, and obtained the nonlinear solution by using the finite element method (FEM) to solve the heat conduction equations. But the solution was only limited in the right angle and cannot be used in complex geometry for the limitation of curvature calculation. Wei and Chen (2011) extended Dong's method to different contact angles and dihedral angles by modifying the curvature calculation algorithm and its relation with the wetting area. They also found some mistakes using FEM to solve the heat conduction equation. After the modifications, the nonlinear solution can be obtained. The proposed theory is validated by experiments of capillary tube and drop tower. Wang (2010) and Xu (2007) studied the influence of initial liquid volume on the capillary flow in interior corner systematically by microgravity experiments using drop tower, including three different conditions: satisfying, close to and dissatisfying the Concus-Finn condition. The experimental results showed that the liquid's front position of the meniscus in the corner increased with the increase of the initial liquid volume.

In many practical applications, the interior corners are rounded instead of ideally sharp due to the manufacturing and processing. Ransohoff and Radke (1988) investigated rounded corners where the rounded portion of the corner was concentric with the free surface. The flow-resistance function was determined numerically for a selection of corner half-angles, contact angles, and degrees of roundedness, which was measured using a ratio that employs the depth of the fluid h . Concus and Finn (1990) found that the critical contact angle predicted by Concus-Finn condition decreased with the increasing of corner radius. Namely, the liquid can form stable interfacial structure easily.

Chen et al. (2006, 2007) set up suitable non-dimensional flow resistance equations with appropriate non-dimensional methods, obtained the approximate analytical solution of capillary driven flow in the rounded interior corner, and verified the results with experiments in drop tower. The results showed that the climbing velocity would be reduced in rounded corner.

The study on the interior corner capillary flow mainly explores the interior corner made up of two straight edges with V-shape. The container is relatively complicated in practical engineering, such as the structure of the actual vane-type surface tension tank (2005) shown in Fig. 1. The capsule is of hemispherical shape in its upper and lower parts, and the intermediate portion is cylindrical. The inside and outside of the vane are fixed on the fixed pole which is in the middle of the tank. The outside of the vane is perpendicular to the wall of the tank. The interior corner formed by the adjacent inside of the vane is V-shaped. Due to the wall of the tank is curved, the interior corner between the wall and the outside of the vane is not V-shaped, which is defined as an interior corner of rounded wall in this article.

In this paper, the capillary driven flow satisfying Concuss-Finn condition in interior corner of rounded wall under microgravity environment has been investigated. Based on Navier-Stokes equations established by Davis (1983) and the capillary driven flow in interior corner of regular polygon researched by Weislogel, the governing equation for capillary driven flow is established and the approximate analytical solution is derived. Moreover the influence of different parameters in interior corner flow is analyzed.

Navier-Stokes Equations and Boundary Conditions of Capillary Flow

The model of capillary flow in an isolated interior corner is shown in Fig. 2. The length of container is L , with straight wall along the radial direction of container and at 45° to y axis. The lowest height of meniscus is H at $z = 0$. H keeps the same as time goes by when the contact angle is less than 30° . Dotted line represents the liquid location at

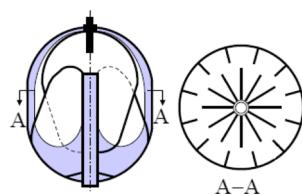


Fig. 1 Schematic model of heavily vanned VTRE tank showing fluid location in low-gravity environment. Tank is partially liquid filled as shown at left with fluid occupying shaded region. Nardin and Weislogel (2005)

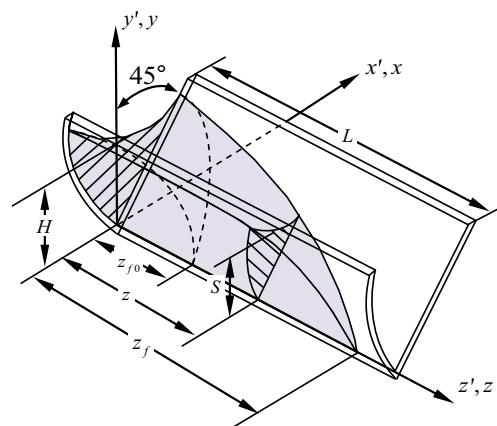


Fig. 2 A fluid column in an isolated interior corner of rounded wall. The coordinate system is aligned such that the z -axis is along the corner. The characteristic height and length of the fluid column are H and L , respectively.

the initial time $t = 0$, z_f and z_{f0} is liquid's front position at any time and initial time, respectively. The cross section of fluid is marked by shaded area at any z -location. x , y , z is non-dimensional rectangular coordinate system, where $x = x'/H$, $y = y'/H$, $z = z'/L$.

Initial meniscus height is an important parameter for the capillary flow in an interior corner of rounded wall under microgravity. Weislogel (1996) found that the capillary flow in V-type interior corner has a constant meniscus height by drop tower experiments, namely initial meniscus height. The capillary flow in an interior corner of rounded wall is numerically simulated using FLOW-3D; the results show that like capillary flow in an interior corner of V-type, the rounded wall also has the initial meniscus height, which is

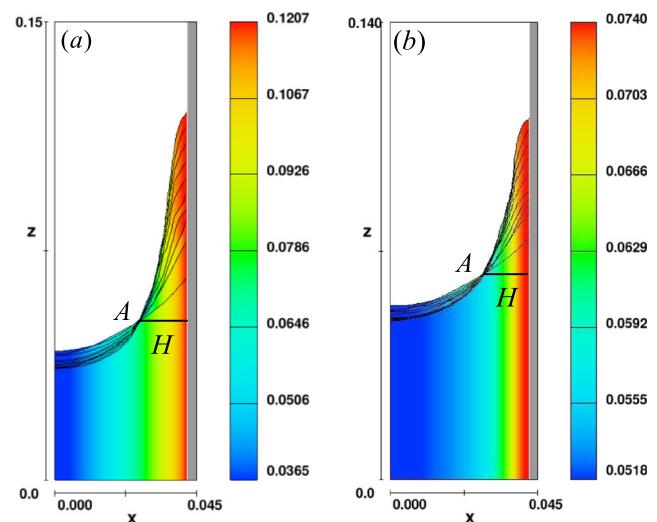


Fig. 3 Capillary rise at different time, which are results from the FLOW-3D simulations. x is the radial direction of the vessel and z is the axial direction. **(a)** $\theta_1 = \theta_2 = 0^\circ$ **(b)** $\theta_1 = 0^\circ, \theta_2 = 10^\circ$

the H as shown in Fig. 3. Figure 3 displays the capillary rise curves at $t = 0.5\text{s}, 1.0\text{s}, 1.5\text{s}, 2.0\text{s}, 2.5\text{s}, 3.0\text{s}, 3.5\text{s}, 4.0\text{s}, 4.5\text{s}, 5.0\text{s}$; and curves have a intersection point A at different time, which corresponds to the initial meniscus height H .

The cross-flow section is sketched in Fig. 4, where θ_1, θ_2 are the contact angles, A and B are intersections between the meniscus and straight, rounded wall, respectively. O_1, O_2 are the corresponding centers for circular curve of A and B, respectively. S is the meniscus height as measured from the x - z plane at a given x_0 ; point C is the intersection point of meniscus AB and the line OO_1 . The minimum distance from point O to meniscus AB is h' , namely $OC = h'$, which is called meniscus height, where the dimensionless is $h = h'/H$. Figure 4 shows that the line OO_1 and y -axis are not coincide, the angle between them is α . If rounded wall OB transforms to straight wall, the interior corner of rounded wall transforms to perfectly sharp interior corner, and the angle $\alpha = 0^\circ$, the line OO_1 and y -axis coincide, namely $x = 0, S_{x=0} = h$. But for rounded wall, $S_{x=0} \neq h$ on $x = 0$ because of angle α . To be convenient, this paper constructed local coordinate system $O\xi\eta$, and set η -axis along the line OO_1 . η -axis divides cross-section into two areas, area I where $\xi > 0$ and area II where $\xi < 0$. The transformation relations between Oxy and $O\xi\eta$:

$$\begin{cases} x = \xi \cos \alpha + \eta \sin \alpha \\ y = -\xi \sin \alpha + \eta \cos \alpha \end{cases} \quad (1)$$

where $\alpha = \arctan(x_1/y_1)$. The linear equation of OA can be expressed as $\eta = k\xi$, in which $k = \tan(\alpha + 45^\circ)$. (x_0, S) can be transformed into (ξ_0, η_0) .

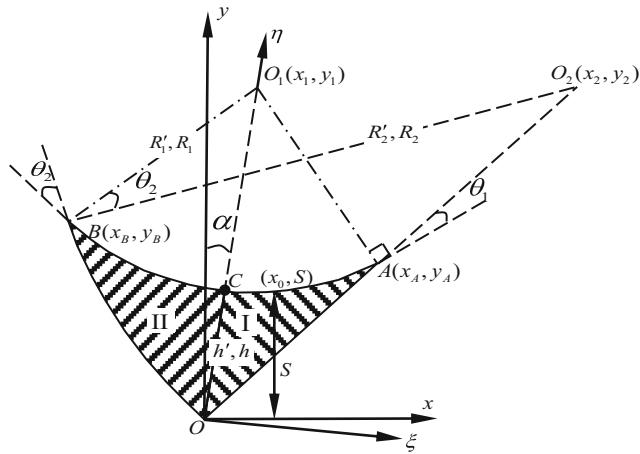


Fig. 4 Schematic diagram of arbitrary cross section. θ_1, θ_2 are the contact angles of straight wall and rounded wall respectively. O_1, O_2 are the corresponding centers of meniscus and rounded wall, R_1, R_2 are the radius respectively. The coordinates of the meniscus are described by x_0 along the x -axis and by S along the y -axis. h is the height of meniscus along OO_1

The non-dimensional Navier-Stokes equations and the continuity equation of capillary flow in interior corner are:

$$\begin{cases} \varepsilon^2 \Theta \frac{Du}{Dt} = -P_{,x} + \varepsilon^2 \nabla^2 u \\ \varepsilon^2 \Theta \frac{Dv}{Dt} = -P_{,y} + \varepsilon^2 \nabla^2 v \\ \Theta \frac{Dw}{Dt} = -P_{,z} + \nabla^2 w \\ \nabla \cdot \mathbf{v} = 0 \end{cases} \quad (2)$$

where $D/Dt = \partial/\partial t + u\partial/\partial x + y\partial/\partial y + w\partial/\partial z$, $\nabla^2 = \partial^2/\partial x^2 + \partial^2/\partial y^2 + \varepsilon^2 \partial^2/\partial z^2$, $\mathbf{v} = (u, v, w)$. u, v, w is dimensionless velocities of x, y, z -component, P is fluid's pressure. $u = u'/\varepsilon W, v = v'/\varepsilon W, w = w'/W, P = HP'/\sigma, W = \varepsilon\sigma/\mu, t = Wt'/L, \Theta = \varepsilon^2\sigma\rho H/\mu^2, \varepsilon = H/L$. μ is the dynamic viscosity, ρ is the density, σ is the surface tension, $P_{,\zeta} = \partial P/\partial \zeta, \zeta = x, y, z$.

According to Weislogel (1996), the boundary condition of capillary flow in interior corner can be described as follows:

(1) No-slip condition along straight wall and rounded wall:

$$u = v = w = 0 \text{ on } y = x \quad (3-a)$$

$$u = v = w = 0 \text{ on } x^2 + y^2 = \sqrt{2}R_2(x + y) \quad (3-b)$$

(2) Meniscus AB is free surface, the stresses are zero:

$$\begin{aligned} & \left(1 + \varepsilon^2 S_{,z}^2\right)^{-1/2} \left(1 + |\nabla S|^2\right)^{-1} \\ & \left\{ \left(1 - S_{,x}^2 + \varepsilon^2 S_{,z}^2\right) [u_{,y} + v_{,x} - S_{,z} (\varepsilon^2 u_{,z} + w_{,x})] \right. \\ & \left. + 2S_{,x} [v_{,y} - u_{,x} - S_{,z} (\varepsilon^2 v_{,z} + w_{,y})] \right. \\ & \left. - \varepsilon^2 S_{,z}^2 (u_{,x} - w_{,z}) \right\} = 0 \end{aligned} \quad (3-c)$$

$$\begin{aligned} & \left(1 + \varepsilon^2 S_{,z}^2\right)^{-1/2} \left(1 + |\nabla S|^2\right)^{-1/2} \\ & \left[\left(1 - \varepsilon^2 S_{,z}^2\right) (\varepsilon^2 v_{,z} + w_{,y}) - S_{,x} (\varepsilon^2 u_{,z} + w_{,x}) \right. \\ & \left. + 2\varepsilon^2 S_{,z} (v_{,y} - w_{,z}) - \varepsilon^2 S_{,x} S_{,z} (u_{,y} + v_{,x}) \right] = 0 \end{aligned} \quad (3-d)$$

where $|\nabla S|^2 = S_{,x}^2 + \varepsilon^2 S_{,z}^2$, $0_{,\zeta} = \partial 0/\partial \zeta$, $0_{,\zeta\chi} = \partial^2 0/\partial \zeta \partial \chi$, $\zeta, \chi = x, y, z$.

(3) The normal stress condition on meniscus AB:

$$\begin{aligned} & -P + 2\varepsilon^2 [1 + |\nabla S|^2]^{-1} \\ & [v_{,y} - S_{,x} (u_{,y} + v_{,x}) - S_{,z} (\varepsilon^2 v_{,z} + w_{,y}) \\ & + S_{,x} S_{,z} (\varepsilon^2 u_{,z} + w_{,x}) + S_{,y}^2 u_{,x} + \varepsilon^2 S_{,z}^2 w_{,z}] \\ & = [1 + |\nabla S|^2]^{-3/2} [S_{,xx} (1 + \varepsilon^2 S_{,z}^2) + \varepsilon^2 S_{,zz} (1 + S_{,x}^2) \\ & - 2\varepsilon^2 S_{,x} S_{,z} S_{,xz}] \end{aligned} \quad (3-e)$$

(4) The angle with rounded wall and the outward unit normal to meniscus is equal to contact angle:

At point A , the contact line boundary condition is:

$$(1 + |\nabla S|^2)^{-1/2} (1 + S_{,x}) = \sqrt{2} \cos \theta_1 \quad (3-f)$$

At point B , the contact line boundary condition is:

$$\frac{1}{R_2} \left(1 + |\nabla S|^2 \right)^{-1/2} \left[\frac{\sqrt{2}}{2} R_2 + S_{,x} \left(x_B - \frac{\sqrt{2}}{2} R_2 \right) - y_B \right] = \cos \theta_2 \quad (3-g)$$

In the process of capillary flow, we can obtain as follows by the z -direction

$$\frac{\partial A}{\partial t} = -\frac{\partial Q'}{\partial z} \quad (4)$$

where A is the cross-sectional area in the x - y plane and Q' is the volumetric flow rate in the z -direction.

Q' can be calculated by the integral of w over the cross-section:

$$Q' = \int_A w dx dy \quad (5)$$

$$R_1 = \frac{-h [h^2 + hR_2 \cos \theta_2 - 2R_2^2] + R_2 h \sqrt{4R_2^2 (\cos^2 \theta_1 + \cos^2 \theta_2) + h (h \sin^2 \theta_1 + 4R_2 \cos \theta_2)}}{2 [(h + R_2 \cos \theta_2)^2 - R_2^2 \sin^2 \theta_1]}$$

Substituting (7-a)–(7-c) into Eq. 6, cross-sectional area A can be expressed by the meniscus height h , namely

$$A = F_A(h) \quad (8)$$

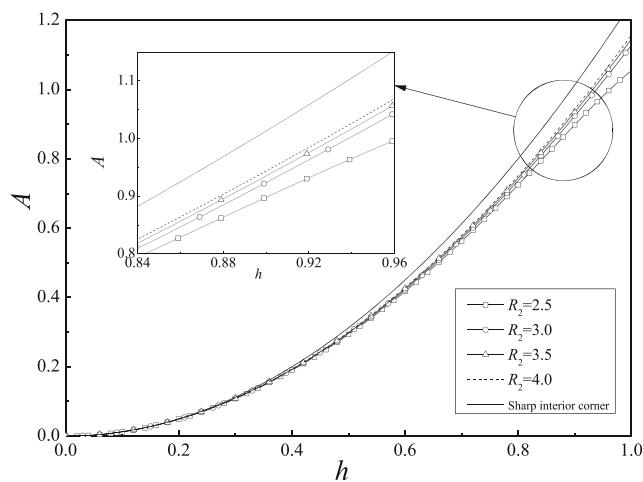


Fig. 5 The analytical relationship between curves of A and meniscus height h with different R_2 when $\theta_1 = \theta_2 = 0^\circ$. h is the height of meniscus along with OO_1 , and R_2 is the radius of the rounded wall. The interior corner becomes sharp as R_2 tends to infinity

According to Fig. 4, cross-sectional area A can be calculated by

$$A = \int_{x_B}^0 \left\{ S - \left[y_2 - \sqrt{R_2^2 - (x - x_2)^2} \right] \right\} dx + \int_0^{x_A} (S - x) dx \quad (6)$$

And the variables S, x_A, x_B in Eq. 6 can be expressed as the function of the meniscus height h , namely

$$S = y_1 - \sqrt{R_1^2 - (x - x_1)^2} \quad (7-a)$$

$$x_A = \sqrt{2} \left[h^2 + 2R_1 h + 2R_1 R_2 (\cos \theta_2 - \sin \theta_1) \right] / (4R_2) \quad (7-b)$$

$$x_B = \left(-b_2 - \sqrt{b_2^2 - 4b_1 b_3} \right) / (2b_1) \quad (7-c)$$

$$\text{where: } b_1 = 4 [(x_2 - x_1)^2 + (y_2 - y_1)^2], b_2 = 4 [(x_2 - x_1) h (h + 2R_1) + 2y_2 (y_2 - y_1) (x_1 - y_1)],$$

$$b_3 = h^2 (h + 2R_1)^2 + 4y_2 h (y_2 - y_1) (h + 2R_1),$$

$$x_2 = y_2 = \sqrt{2} R_2 / 2,$$

$$x_1 = \sqrt{2} \left[h^2 + 2R_1 h - 2R_1 R_2 (\cos \theta_1 - \cos \theta_2) \right] / (4R_2),$$

$$y_1 = \sqrt{2} \left[h^2 + 2R_1 h + 2R_1 R_2 (\cos \theta_1 + \cos \theta_2) \right] / (4R_2),$$

When $\theta_1 = \theta_2 = 0^\circ$, the relationship between curves of cross-sectional area A and meniscus height h with different R_2 as shown in Fig. 5.

When the radius of rounded wall $R_2 \rightarrow \infty$, the rounded wall transforms to the straight wall, the interior corner transforms to the sharp interior corner, where A is positive and proportional to h^2 (Weislogel 1996). Figure 5

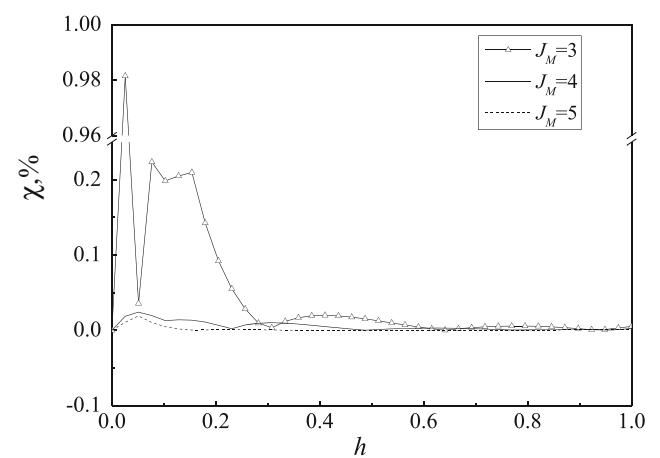


Fig. 6 The relation curves that the relative error χ varies with h , where J_M denotes the highest order in the polynomial

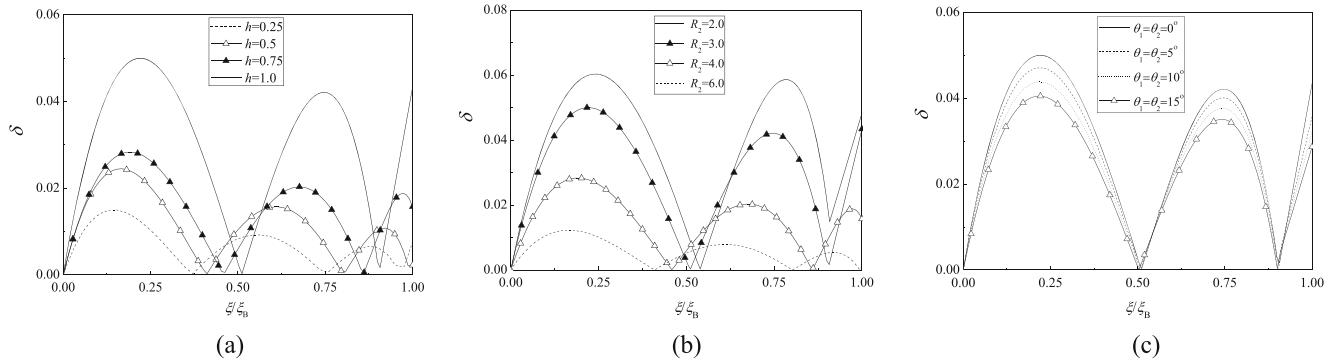


Fig. 7 $\delta \sim \xi/\xi_B$ curves with different structure parameters from analytical results, where ξ_B is the maximum value of ξ . (a) $R_2 = 3.0, \theta_1 = \theta_2 = 0^\circ$ (b) $h = 1.0, \theta_1 = \theta_2 = 0^\circ$ (c) $R_2 = 3.0, h = 1.0$

shows the nonlinear relationship between A and h . And with the increasing of R_2 , it approaches to the relationship in sharp interior corner, namely when R_2 reaches to a certain value, the interior corner of rounded wall can be regarded as sharp interior corner. The error in cross-sectional area between interior corner of rounded wall and sharp interior corner increases with the increasing of h and can be

negligible when $h < 0.4$. It shows that the rounded wall can be simplified as straight wall when $h < 0.4$.

Considering the complication of Eq. 8, cross-sectional area A , for convenience, can be expanded as a function of h

$$A = \sum_{j=1}^{\infty} a_j h^j \quad (9)$$

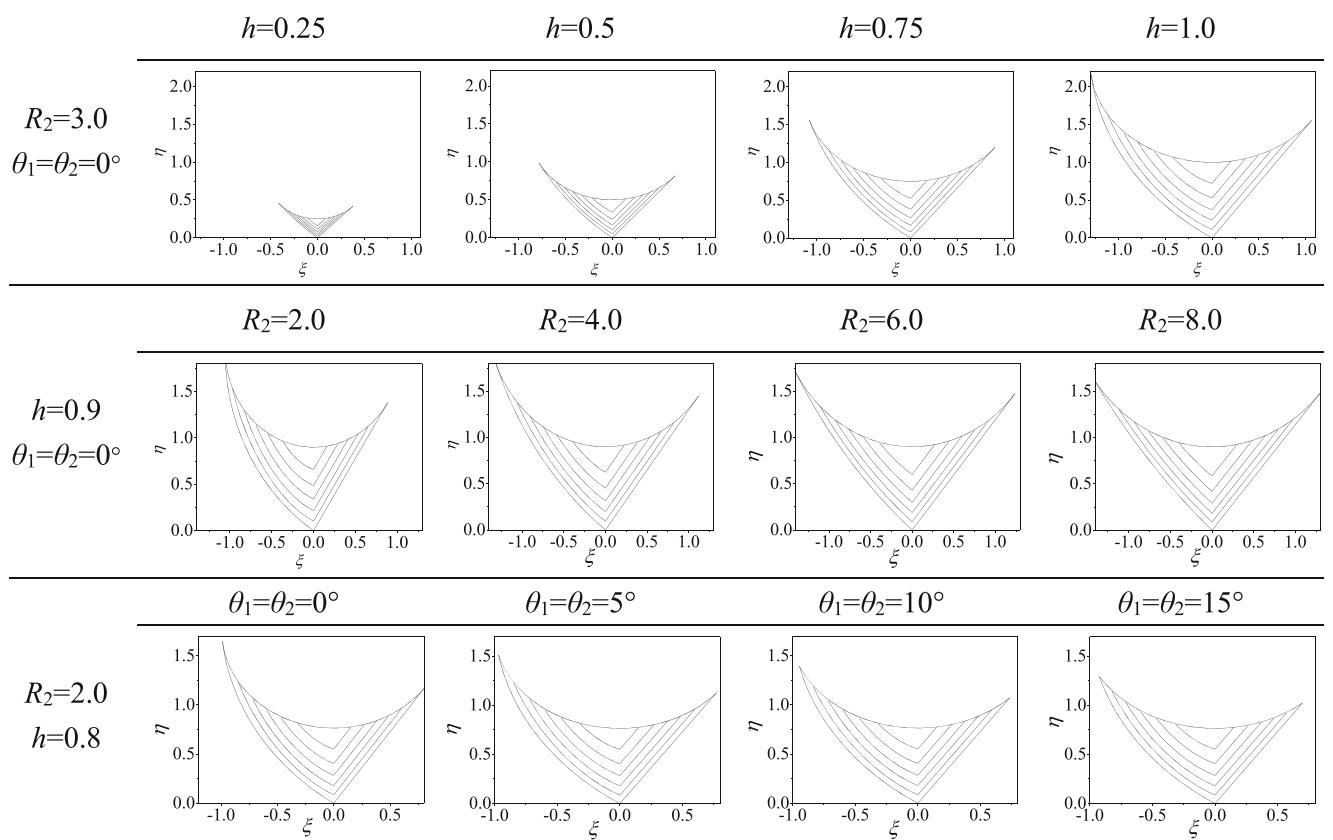


Fig. 8 The analytical solutions of $F_1(\xi, \eta)$ and $F_{II}(\xi, \eta)$ with different structure parameters, where vertical axis and horizontal axis represent ξ and η , respectively. In each graph, right is distribution map of $F_1(\xi, \eta)$, left is distribution map of $F_{II}(\xi, \eta)$

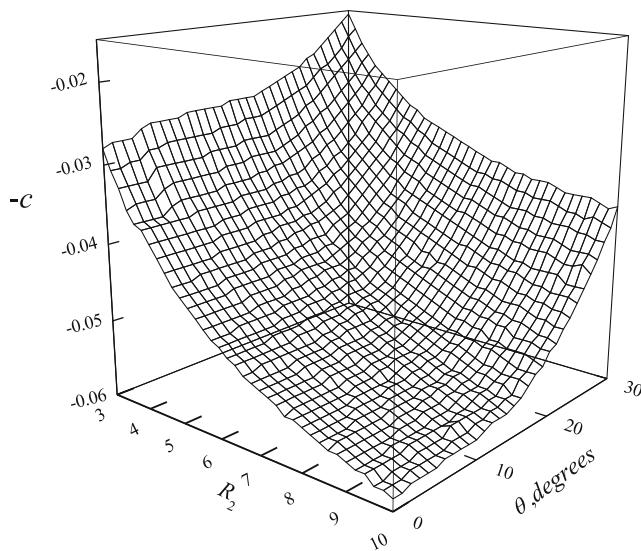


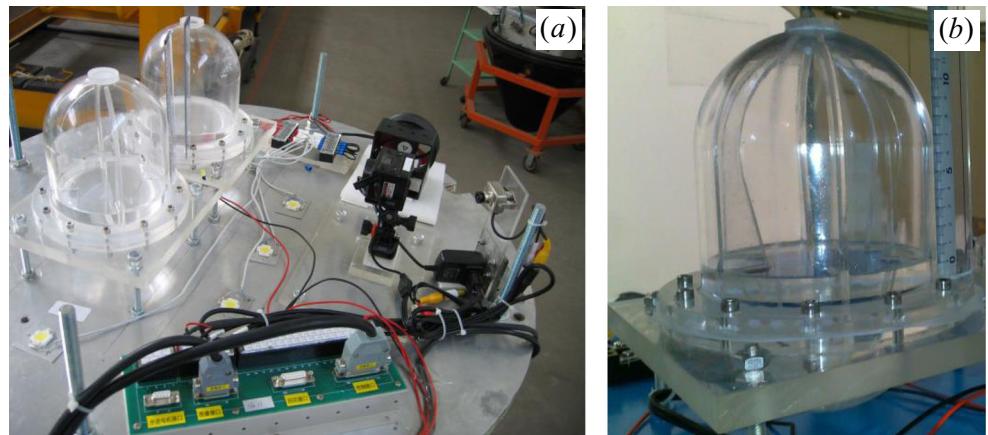
Fig. 9 The surface represents the analytical values of c with the changing contact angles θ and the radius of container R_2

where the coefficients of A by series expansion $a_j (j = 1, 2, \dots, \infty)$ are constants related to R_2, θ_1, θ_2 . For example, when the $R_2 = 3.0, \theta_1 = 0^\circ, \theta_2 = 0^\circ$, the a_j can be obtained that $a_j (j=1,2,3,4)$ equal to 0.0043, 1.2171, 0.0692, -0.0248, respectively, and a_j approach to zero with $j > 4$. The fitting accuracy of area A can be expressed by χ which is the relative error of Eqs. 8 and 9, that

$$\chi = \left| \frac{\sum_{j=1}^{J_M} a_j h^j - F_A(h)}{F_A(h)} \right| \times 100 \%$$

where, J_M is the maximum of j . When $R_2 = 4, \theta_1 = \theta_2 = 0^\circ$, the relation curves that the relative error χ varies with h are obtained in Fig. 6.

Fig. 10 The experimental device, experimental container and the initial position of the liquid (a the experimental device b the experimental container)



The conclusion can be drawn when the highest order of expansion in series equals to 4, the error between the approximate result and original A has become small from Fig. 6. So the non-dimensional area A can be expanded as the four order polynomial about h .

Dynamic Equation and Approximate Analytical Solution of Capillary Flow

The perturbation method is used to solve the Navier-Stokes equations of capillary flow

$$\begin{cases} u = u_0 + \varepsilon^2 u_1 + \dots \\ v = v_0 + \varepsilon^2 v_1 + \dots \\ w = w_0 + \varepsilon^2 w_1 + \dots \\ P = P_0 + \varepsilon^2 P_1 + \dots \end{cases}, \quad (10)$$

In this research, the length of container $L \gg H$, we can assume that $\varepsilon^2 \ll 1$. Substituting (10) into Eq. 2, (3), where terms of $O(\varepsilon)$ are absorbed into the leading order terms for convenience. The Navier-Stokes equations can be simplified as:

$$P_{0,z} = w_{0,xx} + w_{0,yy} \quad (11)$$

The boundary conditions (3-a)-(3-g) may be simplified as follows; and Eqs. 12-d and 12-e represent the boundary conditions on the interface.

$$w_0 = 0 \text{ on } y = x \quad (12-\text{a})$$

$$w_0 = 0 \text{ on } x^2 + y^2 = \sqrt{2}R_2(x + y) \quad (12-\text{b})$$

$$\begin{aligned} (1 - S_{,x}^2)(u_{0,y} + v_{0,x} - S_{,z}w_{0,x}) \\ + 2S_{,x}(v_{0,y} - u_{0,x} - S_{,z}w_{0,y}) = 0 \end{aligned} \quad (12-\text{c})$$

$$w_{0,y} - S_{,x}w_{0,x} = 0 \quad (12-\text{d})$$

$$-P_0 = S_{,xx}^2(1 + S_{,x}^2)^{-3/2} \quad (12-\text{e})$$

Table 1 Properties of the liquids

Fluid	$v/(\text{cm/s})$	$\sigma /(\text{N/m} \times 10^{-3})$	$\rho /(\text{kg} \cdot \text{m}^{-3} \times 10^3)$	$\mu/(\text{pa} \cdot \text{s} 10^{-3})$	$\theta(^{\circ})$
Silicone oil	5	18.5	0.915	4.575	0
	10	19.32	0.935	9.350	0

$$1 + S_{,x} = \sqrt{2} \cos \theta_1 \left(1 + S_{,x}^2\right)^{1/2} \quad (12-f)$$

$$\begin{aligned} \frac{1}{R_2} \left(1 + |\nabla S|^2\right)^{-1/2} & \left[y_B - \frac{\sqrt{2}}{2} R_2 - S_{,x} \left(x_B - \frac{\sqrt{2}}{2} R_2\right) \right] \\ & = \cos \theta_2 \end{aligned} \quad (12-g)$$

Substituting (7-a) into Eq. 12-d:

$$w_{0,y} = \frac{x - x_1}{y_1 - S} w_{0,x} \quad (13)$$

Substituting (7-a) into Eq. 12-e:

$$P_0 = -R_1^{-1} \quad (14)$$

Due to the radius of meniscus R_1 is a function of z, t , so is P_0 . Then the $w_{0,xx} + w_{0,yy}$ should not contain x, y from Eq. 11. Considering the transformation relations in Eq. 1, flow velocity w_0 can be set as a quadratic function of x and y . The area I in the Fig. 4 can be set as follows:

$$w_0 = w_{0I}(\xi, \eta) = F_I(\xi, \eta) \frac{\partial P_0}{\partial z} \quad (15)$$

where $F_I(\xi, \eta) = C_{I,1}\xi^2 + C_{I,2}\eta^2 + C_{I,3}\xi\eta + C_{I,4}\xi + C_{I,5}\eta + C_{I,6}$.

Equation 15 must satisfy (11) and boundary conditions (12-a) and Eq. 13, so

$$\begin{aligned} C_{I,1} &= \frac{k^2}{2(1+k^2)}, \quad C_{I,2} = \frac{1}{2(1+k^2)}, \\ C_{I,3} &= -\frac{k}{1+k^2}, \quad C_{I,4} = \frac{(\eta_0 - \xi_0 k)k}{1+k^2}, \\ C_{I,5} &= \frac{\xi_0 k - \eta_0}{1+k^2}, \quad C_{I,6} = 0 \end{aligned} \quad (16)$$

The area II can be set as

$$w_0 = w_{0II}(\xi, \eta) = F_{II}(\xi, \eta) \frac{\partial P_0}{\partial z}. \quad (17)$$

Equation 17 to satisfy the Eq. 11 and boundary condition (13) under the premise, $w_{0II}(\xi, \eta)$ can be set as a quadratic function of ξ and η , so

$$F_{II}(\xi, \eta) = C_{II,1}\xi^2 + C_{II,2}\eta^2 + C_{II,3}\xi\eta + C_{II,4}\xi + C_{II,5}\eta + C_{II,6}$$

Because of the continuity of flow, $w_{0I}(\xi, \eta)$ equals to $w_{0II}(\xi, \eta)$ along OC . For $\xi = 0$, $w_{0I}(0, \eta) = w_{0II}(0, \eta)$. Equation 17 must satisfy (11) and Eq. 13, so

$$C_{II,1} = \frac{k^2}{2(1+k^2)}, \quad C_{II,2} = \frac{1}{2(1+k^2)},$$

$$\begin{aligned} C_{II,3} &= \frac{\eta_1 - \eta_0}{\xi_0(\eta_1 - 2\eta_0)} \left(2C_{II,1} \frac{\xi_0^2}{\eta_1 - \eta_0} - 2C_{II,2}\eta_0 - C_{II,5} \right) \\ &+ \frac{C_{II,4}}{\eta_1 - 2\eta_0}, \quad C_{II,5} = \frac{\xi_0 k - \eta_0}{1+k^2}, \quad C_{II,6} = 0 \end{aligned} \quad (18)$$

Because the boundary condition of the interface between the two areas of the corner has a higher priority than the boundary condition along the rounded wall; and Eq. 17 can only be ξ and η of the bivariate quadratic function, leads to the fact that it is impossible to find solutions for $w_{0II} = 0$ along the rounded wall. To minimize overall residual, the specific value of $C_{II,4}$ can be obtained by least square method which is described as

$$G = \int_{OB} \left(C_{II,1}\xi^2 + C_{II,2}\eta^2 + C_{II,3}\xi\eta + C_{II,4}\xi + C_{II,5}\eta - 0 \right)^2 d\eta \quad (19)$$

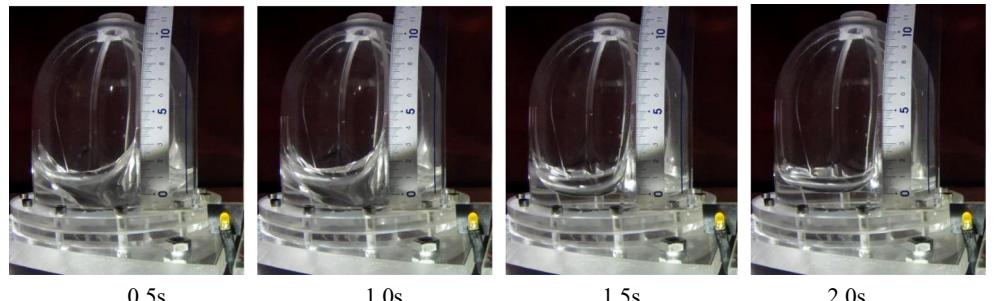
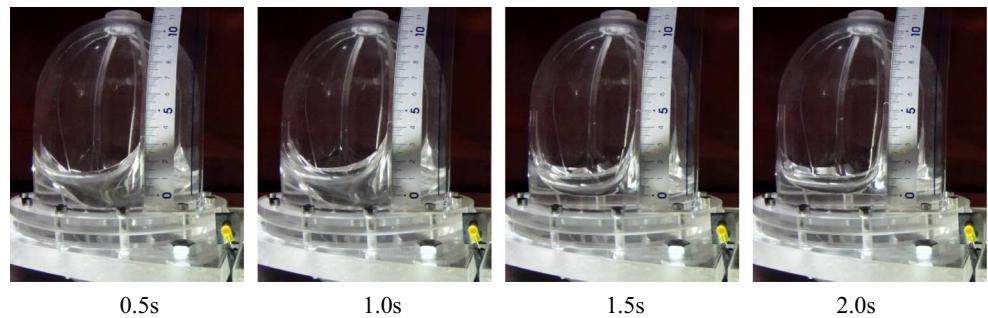
Fig. 11 The liquid level scattergram of 5cs fluid at different moments

Fig. 12 The liquid level scattergram of 10cs fluid at different moments



The $C_{II,4}$ can be derived by $dG/dC_{II,4} = 0$. The relative error δ of $w_{0II}(\xi, \eta)$ in the rounded wall OB can be expressed as

$$\delta = \left| \frac{w_{0II}(\xi, \eta)|_{OB} - 0}{w_{0II}(\xi, \eta)_{max}} \right| \times 100 \%,$$

where, $w_{0II}(x\xi, \eta)|_{OB}$ is values of $w_{0II}(\xi, \eta)$ along the rounded wall OB , $w_{0II}(\xi, \eta)_{max}$ is the maximum value of $w_{0II}(\xi, \eta)$ in the region II. The point B will change with different h , R_2, θ_1 , consequently, the relative error δ will change. To be convenient, when we explore the relationship between δ and h , R_2 and θ_1 will be fixed. In following, δ with different parameters h, R_2, θ_1 are shown in Fig. 7.

Figure 7 shows that, the value of the relative error δ is relatively little with different structural parameters, and the maximum value is near 0.06 %. The relative error δ of $w_{0II}(\xi, \eta)$ increases with the increasing of h when R_2, θ_1 and θ_2 are fixed, decrease with increasing of R_2 when h, θ_1, θ_2 are fixed and increasing of $\theta_1(\theta_2)$ when R_2, h are fixed.

Diagram of analytical solution of $F_I(\xi, \eta)$ and $F_{II}(\xi, \eta)$ with different structure parameters as shown in Fig. 8.

Figure 8 shows that, the wetting length along rounded wall is longer than that along the straight wall when the contact angles are equal on both walls, and the distribution area of $F_{II}(\xi, \eta)$ is larger than that of $F_I(\xi, \eta)$. The distribution areas of $F_I(\xi, \eta)$ and $F_{II}(\xi, \eta)$ decrease with the decreasing of h when R_2, θ_1 and θ_2 are fixed. When $h \rightarrow 0$, the distribution areas of $F_I(\xi, \eta)$ and $F_{II}(\xi, \eta)$ approach zero, which lead to the liquid's tip. The wetting length along rounded wall and distribution area of $F_{II}(\xi, \eta)$ decrease with

the increasing of R_2 when h, θ_1 and θ_2 are fixed. The wetting length along rounded wall closes to the wetting length along the straight wall, and $F_I(\xi, \eta)$, $F_{II}(\xi, \eta)$ tend to be symmetrical figure gradually. When R_2, h are fixed and $\theta_1 = \theta_2$, the wetting length along walls and distribution areas of $F_I(\xi, \eta)$ and $F_{II}(\xi, \eta)$ decrease with increases of θ_1 , and the decrease rate of $F_{II}(\xi, \eta)$ is greater than that of $F_I(\xi, \eta)$.

Substituting (14), (15) and Eq. 17 into Eq. 5, the volumetric flow rate Q' can be expressed as

$$Q' = \frac{1}{R_1^2} \frac{\partial R_1}{\partial z} \left(\int_{A_I} F_I d\xi d\eta + \int_{A_{II}} F_{II} d\xi d\eta \right) \quad (20)$$

When R_2 and θ_1, θ_2 are fixed, Q' can be fitted as a function of h , namely

$$Q' = \sum_{i=1}^{\infty} b_i h^i \frac{\partial h}{\partial z} \quad (21)$$

where $b_i (i = 1, 2, \dots, \infty)$ are constants. Substituting (9), (21) into Eq. 4, the dynamic equation of capillary flow in interior corners of rounded wall can be obtained:

$$\sum_{j=1}^{\infty} a_j j h^{j-1} \frac{\partial h}{\partial t} + \sum_{i=1}^{\infty} b_i i h^{i-1} \left(\frac{\partial h}{\partial z} \right)^2 + \sum_{i=1}^{\infty} b_i h^i \frac{\partial^2 h}{\partial z^2} = 0 \quad (22)$$

Considering (22) is the nonlinear partial differential equation, we transform it to the nonlinear ordinary differential equations through introducing an auxiliary variable. The h can be set as a function of λ :

$$h = h(\lambda) \quad (23)$$

Fig. 13 The partial enlarged view in Fig. 11 of the liquid's front position

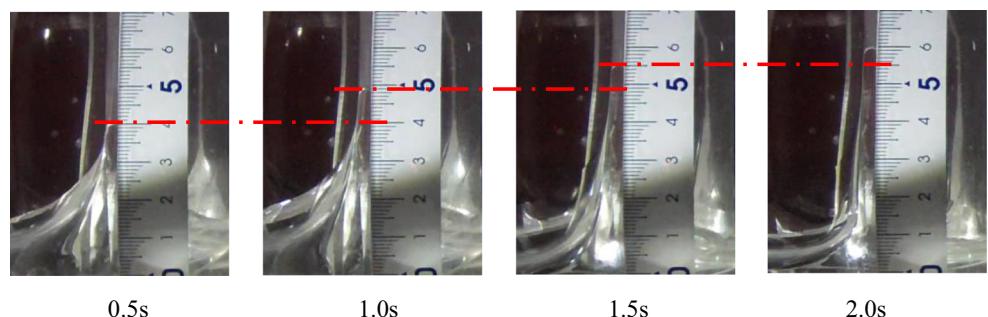
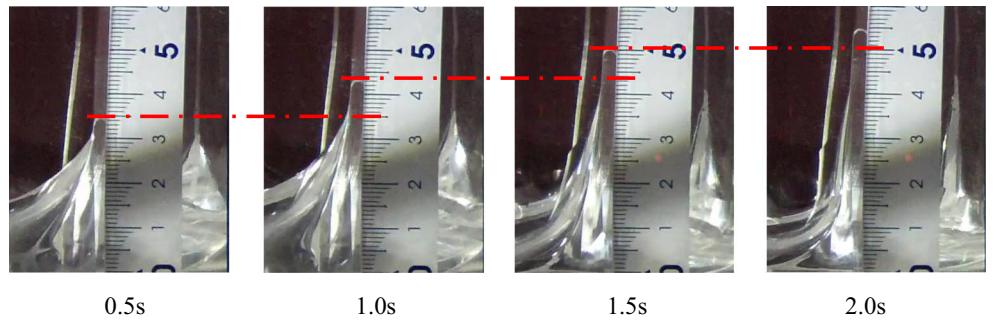


Fig. 14 The partial enlarged view in Fig. 12 of the liquid's front position



where $\lambda = z/z_f$, $\lambda \in [0,1]$. Then Eq. 22 can be transformed as

$$\begin{aligned} & -\sum_{j=1}^{\infty} a_j j h^{j-1}(\lambda) \lambda z_f \dot{z}_f \frac{dh(\lambda)}{d\lambda} + \sum_{i=1}^{\infty} b_i i h^{i-1}(\lambda) \left[\frac{dh(\lambda)}{d\lambda} \right]^2 \\ & + \sum_{i=1}^{\infty} b_i h^i(\lambda) \frac{d^2 h(\lambda)}{d\lambda^2} = 0 \end{aligned} \quad (24)$$

where $\dot{z}_f = dz_f/dt$. Equation 24 is a function of λ except for term $z_f \dot{z}_f$, so $z_f \dot{z}_f$ must be a constant:

$$z_f \dot{z}_f = c \quad (25)$$

where c is a constant. $z_f(t)$ can be obtained from Eq. 25

$$z_f = \sqrt{2ct + z_{f0}^2} \quad (26)$$

where z_{f0} is the liquid's front position at initial time as shown in Fig. 2. $h(\lambda)$ can be solved by series solution and can be set as:

$$h(\lambda) = \sum_{k=0}^{\infty} d_k \lambda^k \quad (27)$$

Since $h = 1$ when $\lambda = 0$ and $h = 0$ when $\lambda = 1$, then

$$\begin{cases} d_0 = 1 \\ \sum_{k=0}^{\infty} d_k = 0 \end{cases} \quad (28)$$

Noting that $z_f \dot{z}_f = c$, and substituting (27) into Eq. 24

$$\begin{aligned} & -c \sum_{j=1}^{\infty} a_j j \lambda \left(\sum_{k=0}^{\infty} d_k \lambda^k \right)^{j-1} \left(\sum_{k=1}^{\infty} d_k k \lambda^{k-1} \right) \\ & + \sum_{i=1}^{\infty} b_i i \left(\sum_{k=0}^{\infty} d_k \lambda^k \right)^{i-1} \left(\sum_{k=1}^{\infty} d_k k \lambda^{k-1} \right)^2 \\ & + \sum_{i=1}^{\infty} b_i \left(\sum_{k=0}^{\infty} d_k \lambda^k \right)^i \left[\sum_{k=2}^{\infty} d_k k(k-1) \lambda^{k-2} \right] = 0 \end{aligned} \quad (29)$$

Equation 29 is an algebraic equation about λ , which can be obtained by setting the coefficients in front of λ^k ($k = 1, 2, \dots, \infty$) to zero, namely

$$f_k(d_1, d_2, \dots) = 0, k = 1, 2, \dots \quad (30)$$

Equation 29 contains an unknown constant c that can be solved iteratively by making the volume of liquid capillary flow at any time t is equal to the integral of the volumetric flow rate Q' at the $z = 0$ on $[0, t]$, that is

$$\int_0^{z_f} Adz - \int_0^{z_{f0}} A|_{t=0} dz = \int_0^t Q'|_{z=0} dt \quad (31)$$

Substituting (9), (21) and Eq. 27 into Eq. 31

$$\begin{aligned} & \int_0^{z_f} \left\{ \sum_{j=1}^{\infty} a_j \left[\sum_{k=0}^{\infty} d_k (z/z_f)^k \right]^j \right\} dz \\ & - \int_0^{z_{f0}} \left\{ \sum_{j=1}^{\infty} a_j \left[\sum_{k=0}^{\infty} d_k (z/z_f)^k \right]^j \right\} dz \\ & = \left(\sum_{i=1}^{\infty} b_i \right) d_1 (z_f - z_{f0}) / c \end{aligned} \quad (32)$$

Make

$$\sum_{j=1}^{\infty} a_j \left[\sum_{k=0}^{\infty} d_k (z/z_f)^k \right]^j = \sum_{m=0}^{\infty} g_m (z/z_f)^m \quad (33)$$

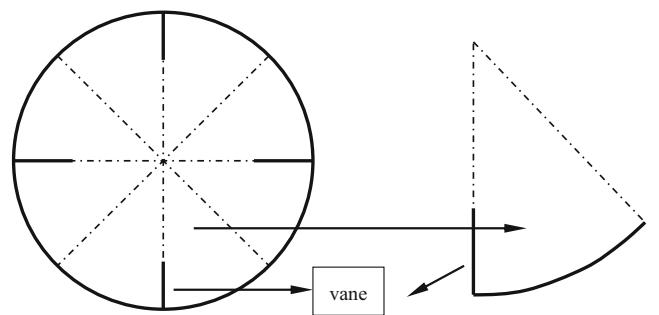
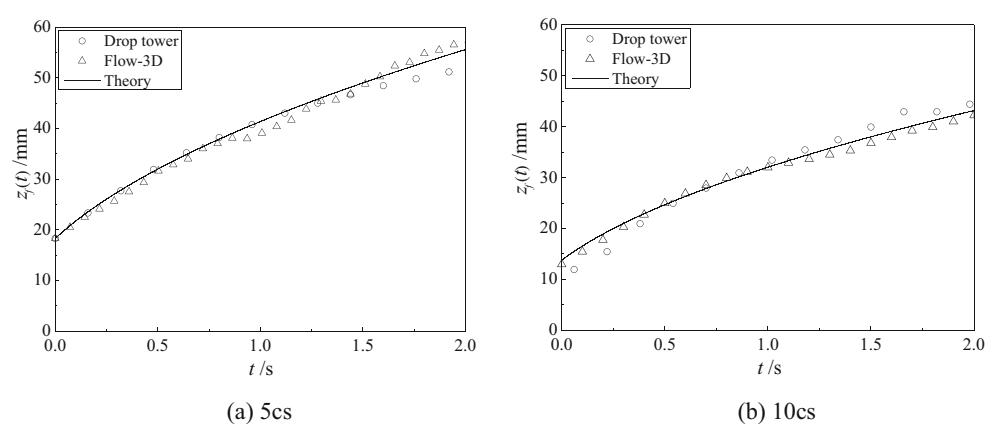


Fig. 15 The cross sections of tank and 1/8 tank. Dash lines represent liquid-liquid contact, and the contact angle is 90°

Fig. 16 The comparisons of the liquid's front position



then Eq. 32 can be simplified as

$$\sum_{m=0}^{\infty} \frac{g_m}{m+1} = \left(\sum_{i=1}^{\infty} b_i \right) d_1 / c \quad (34)$$

The values of c and d_k ($k = 1, 2, \dots$) can be obtained by solving (30) and Eq. 34 iteratively, and then h can be obtained. As is shown in Fig. 9, the relationship between c and variables R_2 and θ ($\theta_1 = \theta_2 = \theta$) can be obtained by changing the value of R_2 and θ with the boundary condition in Eqs. 12-a–12-e. In Fig. 9, c increases with increasing R_2 when θ is fixed, and decreases with increasing θ when R_2 is fixed. Combining with Eq. 26, z_f would decrease with increasing θ when other parameters are fixed.

The Microgravity Drop Tower Verification

Microgravity experiments were conducted to verify the correctness of the proposed method. The experiments investigated the time evolution of the liquid's front. The experimental device and experimental container are shown in Fig. 10.

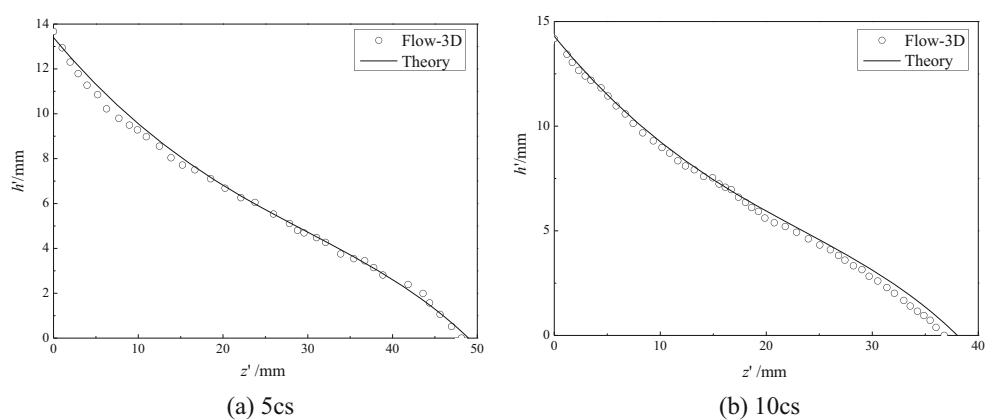
The experimental container was cylindrical with the radius of 60 mm and made of PMMA; it contains four vanes

which were perpendicular to the wall of the tank. Due to radius of the chosen rounded container in experiment being much longer than that of meniscus, the effect of the other interior corner can be neglected. The experimental mediums are silicone oil fluids. Physical properties of the medium are shown in Table 1. The liquid's front position can be observed through gauges that fixed beside the experimental container. Short-term microgravity experiments can be produced using microgravity drop-tower of key laboratory in the Chinese academy of sciences, the gravity level was 10^{-3} g. The distributions of liquid level in a different time are shown in Figs. 11, 12, 13 and 14. And the pixels of the observed pictures from Figs. 11 to 14 are 1920*1080.

The FLOW-3D software was used to explore the numerical analysis of this study. With FAVOR and improved finite difference methods, the FLOW-3D software is appropriate for the models of steady/unsteady flow, Newtonian fluid/non-Newtonian fluid, free surface, porous media, et al. Different from other computational fluid dynamics software, the FLOW-3D increases the accuracy of the tracking of free surface and can be used to accurately simulate the capillary driven flow in an interior corner.

The cross section of the tank is as shown in Fig. 15. To simplify the model and shorten the calculation time in the CFD simulations, 1/8 tank has been modeled, where

Fig. 17 Meniscus height h' ($h' = h^*H$) in different cross-section with coordinate z' at $t = 1.5$ s



the dash lines represent liquid-liquid contact, and the contact angle is 90° . After computing many grids, the results have a better convergence and stability when the grids are $100 \times 100 \times 300$.

As shown in Fig. 16, theoretical calculation results of the liquid's front position z_f are compared with those from drop-tower experiments and numerical simulation using FLOW-3D software.

Figure 16 shows that the liquid's front positions of theoretical calculation results are consistent with experiments and numerical results from the FLOW-3D. We can know from Eq. 26 that the liquid's front position of capillary flow in cylindrical interior corners is proportional to $t^{1/2}$, which corresponds with the rule of capillary flow in prismatic interior corners.

When the liquid's front position z_f is determined, we can calculate the meniscus height according to Eq. 27. The

comparisons of theoretical calculation of $h'(h' = h^*H)$ and numerical results are shown in Fig. 17 when $t = 1.5\text{s}$.

As shown in Fig. 17, the meniscus height of theoretical calculation is consistent with numerical simulation results using the FLOW-3D, verifying the correctness of theoretical calculation.

The Influence of Different Structure Parameters on the Capillary Flow

Figure 18 shows the relationship curves between non-dimensionalized meniscus height h and λ when the radius of rounded wall R_2 and contact angles θ_1, θ_2 are selected in different values. The liquid is 5cs fluid.

As shown in Fig. 18a, when $\theta_1 = \theta_2 = 0^\circ$, the meniscus height h increases with the radius of rounded wall

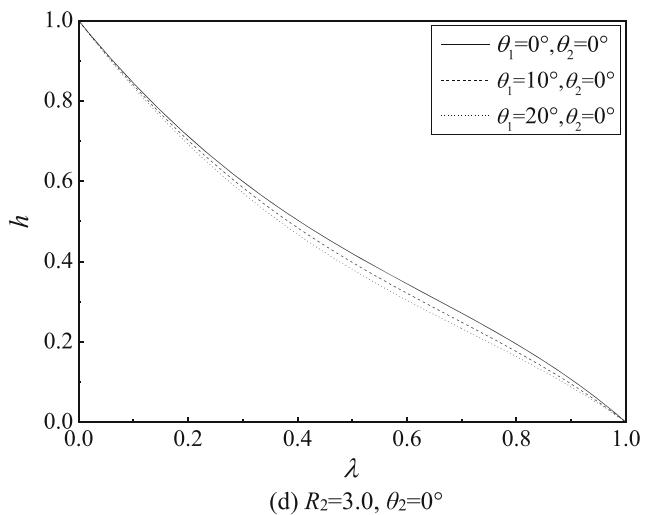
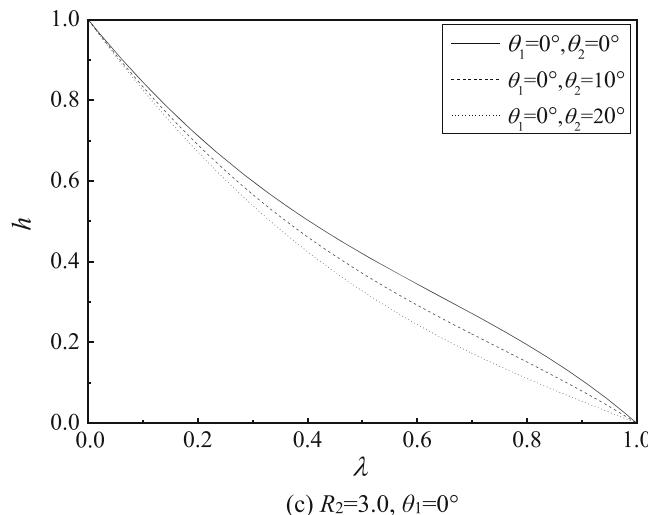
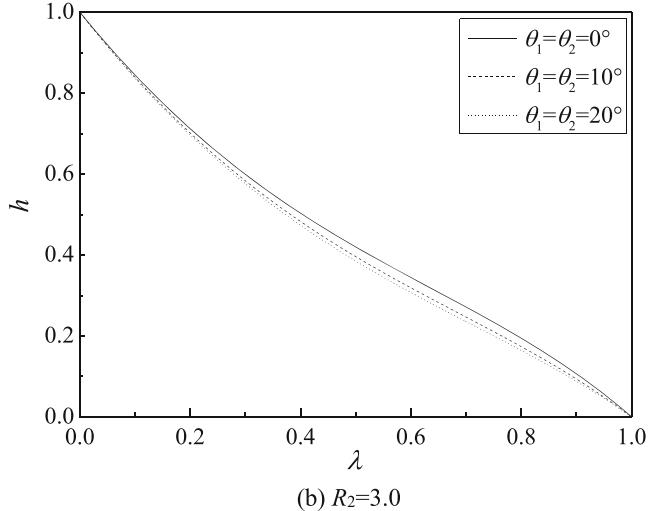
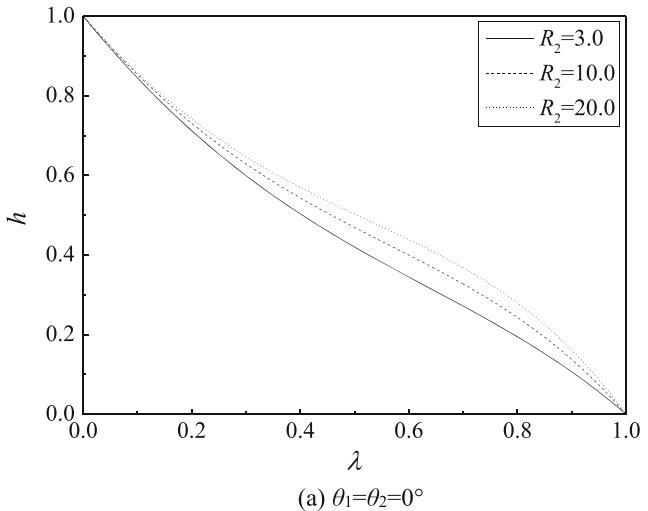


Fig. 18 The curves of $h - \lambda$ with different structure parameters which are the results from approximate analytical solution

R_2 increasing. When $R_2 \rightarrow \infty$, the rounded wall can be regarded as straight wall, namely the meniscus height of rounded interior corner is smaller than that of perfectly sharp interior corner; Fig. 18b shows that the smaller contact angle, the higher meniscus height would be with fixed R_2 . Figure 18c shows the effect of θ_2 on h with fixed R_2 and θ_1 ; Fig. 18d shows the effect of θ_1 on h with fixed R_2 and θ_2 . It shows that, the meniscus height h decreases with the increase of contact angle, but the influence of decrease of the contact angle on rounded wall is greater than that on the straight wall (see Figs. 18c and 18d). The results can be applied in the space fluid management. For example, if you want to keep liquid surface smooth in the container, the climbing height of capillary flow in cylindrical interior corners should be reduced, which means that you should choose a liquid or material with a large contact angle along rounded wall. The influence of θ_2 on h when θ_2 changes solely is greater than that when θ_1, θ_2 change together, which means the influence of θ_1 on h reduces the effect of θ_2 when θ_1, θ_2 change together (see Figs. 18b and 18c). The influence of θ_1 on h when θ_1 changes solely is almost the same as θ_1, θ_2 change together (see Figs. 18b and 18d).

Conclusions

In this paper, the capillary flow in cylindrical interior corners in a microgravity environment has been investigated under the Concus-Finn condition. The governing equation of capillary driven flow in cylindrical interior corners has been established, and the approximate analytical solution has been obtained. The function that liquid's front position is proportional to $t^{1/2}$ is derived. The influence of different parameters on the interior corner flow is explored by using a set of typical parameters. The results show that the non-dimensional meniscus height h decreases with increasing contact angle, while increases with increasing radius of rounded wall. And the influence of decrease of the contact angle on rounded wall is greater than that on straight wall. When contact angles of the two walls change together, the influence of the contact angle on straight wall inhibits that on rounded wall. Our work can be applied to design containers and choose the suitable solution in the space fluid management. For example, if you want to keep liquid

surface smooth in the container, the climbing height of capillary flow in cylindrical interior corners should be reduced, which means that you can choose liquids or materials with a large contact angle along rounded wall.

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