

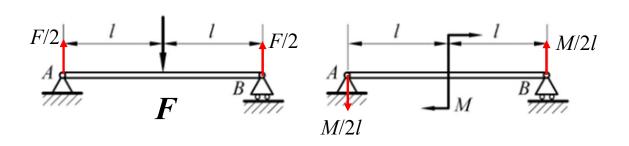
### 上节课内容回顾

力矩

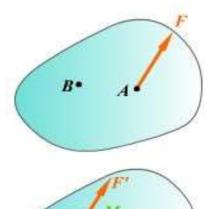


力与力偶是两个独立的静力学要素: 力偶是一对等值方向不共线的平行力

1. 力只能由力平衡,力偶只能由力偶平衡



2. 力的平移定理





-因为力与力偶独立,所以 平移力只能产生力偶





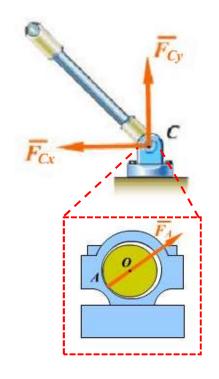




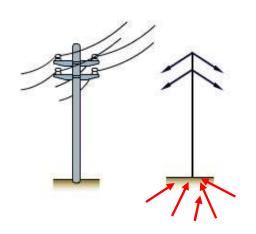


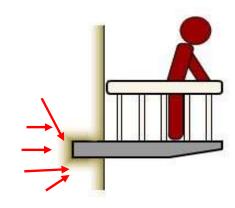


### 平面固定端约束: 平面任意力系向一点简化实例



平面固定铰链支座 约束位移,自由旋转





简化结果一般为主矢、主矩均不为0



约束力系的主矢约束平面运动,主矩约束转动



不能移动+不能旋转



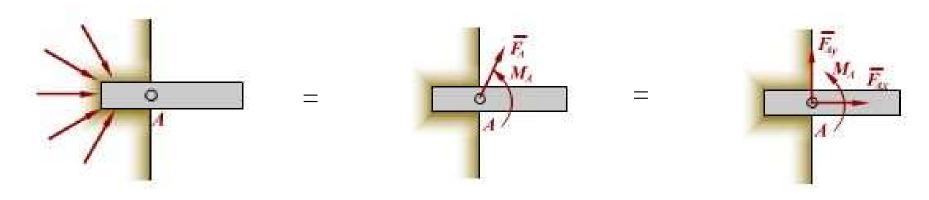


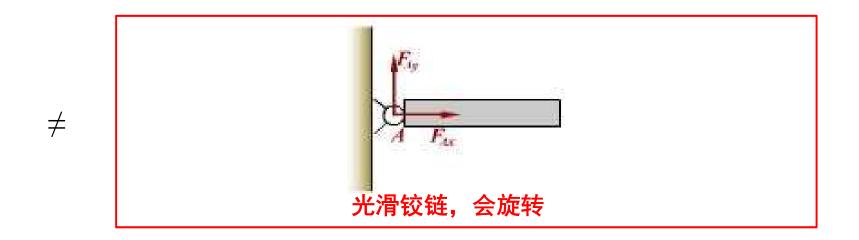






### 固定端的约束力为平面任意力系往一点简化的结果









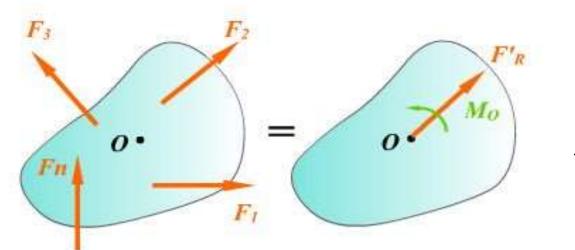








### 三. 平面任意力系的简化结果分析



几个简化结果?

$$(1) \ \overline{F}'_{R} \neq 0 \quad M_{O} \neq 0$$



可以继续简化,为什么?

$$(2) \ \overline{F}'_{R} \neq 0 \quad M_{O} = 0$$

变为情况(2)

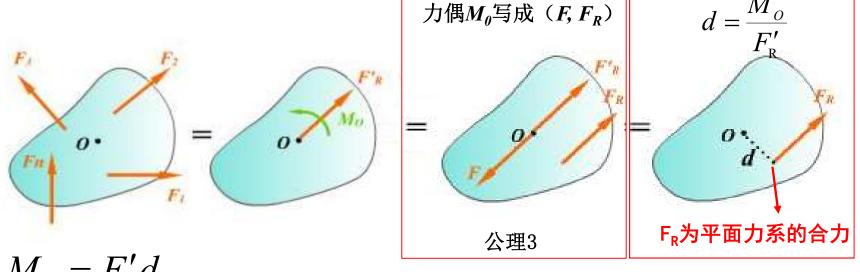
(3) 
$$\overline{F}'_{R} = 0$$
  $M_O \neq 0$ 

(4) 
$$\bar{F}'_{R} = 0$$
  $M_{O} = 0$ 





$$\overline{F}'_{R} \neq 0$$
  $M_{o} \neq 0$   $\longrightarrow$   $F_{R}$ 为平面任意力系的合力,作用线离简化中心 $O$ 距离为 $M_{o}/|F'_{R}|$ 



$$M_O = F_R'd$$
 
$$F_R = F_R' = F$$
 合力对 $O$ 的力矩 力系对 $O$ 的力矩

合力矩定理: 平面任意力系的合力对作用面内任一点的矩等于力系中各力对同一点的矩代数和

(平面力系往一点简化成合力,则主矩为0)。

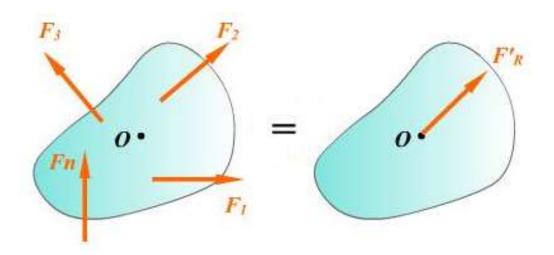


### 三. 平面任意力系的简化结果分析

$$\overline{F}'_{R} \neq 0$$
  $M_{o} = 0$  一 在 $o$ 点的主矩为 $0$ 

合力:平面力系向作用面内点简化后主矩为0,对 应的主矢为平面力系的合力

### --平面力系可以简化为一个合力





### 例2-9(分布力力矩计算)

已知: q,l;

求: 合力及合力作用线位置.

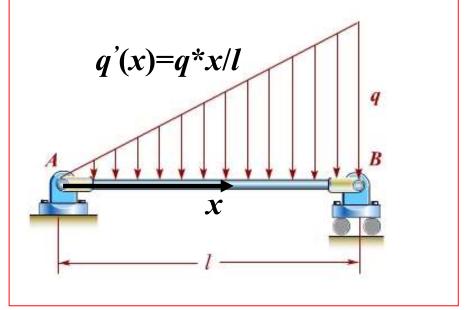
### 解: 建立坐标系

微元dx上的力为:

$$q'dx = \frac{x}{l} \cdot qdx$$

选取A为简化中心,

$$F_{R} = \int_{0}^{l} \frac{x}{l} \cdot q \cdot \mathbf{d}x = \frac{1}{2}ql$$
 主矢
$$M_{A} = \int_{0}^{l} \frac{x}{l} \cdot q \cdot x \cdot \mathbf{d}x = \frac{1}{3}ql^{2}$$
 主矩



往A点简化主矢、主矩均不为0

合力的作用点的主矩为0











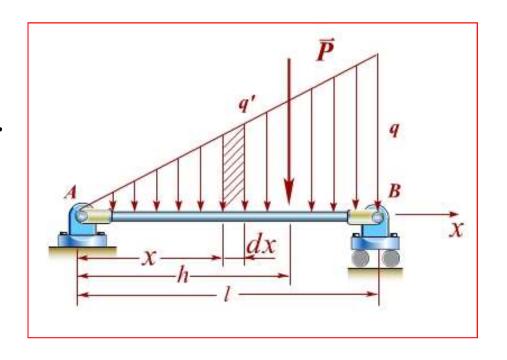
### 例2-9(分布力力矩计算)

已知: q,l;

求: 合力及合力作用线位置.

解: 合力大小为主矢大小

$$P = F_R = \int_0^l \frac{x}{l} \cdot q \cdot dx = \frac{1}{2} q l$$



#### 合力矩定理:

合力对固定点的力矩等于分力对固定点的力矩

$$P \cdot h = \int_{0}^{l} q' \cdot x \, dx = \int_{0}^{l} \frac{x^{2}}{l} q \, dx = \frac{ql^{2}}{3}$$

因为 
$$P = \frac{1}{2}ql$$
 得  $h = \frac{2}{3}l$ 













### 分布力: 平面平行力系

主矢:向A点简化

$$F_{RA} = \int_{0}^{l} \frac{x}{l} \cdot q \, \mathbf{d}x = \frac{1}{2} q l$$

主矩:向A点简化

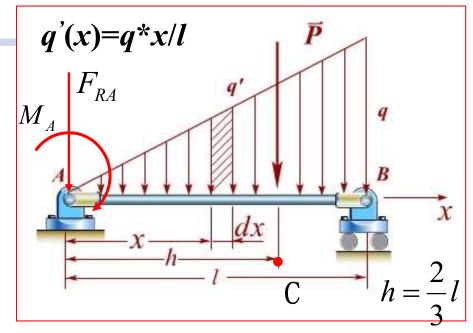
$$M_A = \int_0^l q' \, \mathbf{d}x \cdot x = \int_0^l \frac{x^2}{l} q \, \mathbf{d}x = \frac{ql^2}{3}$$

主矢:向C点简化

$$F_{RC} = \int_{0}^{l} \frac{x}{l} \cdot q \, \mathbf{d}x = \frac{1}{2} q l$$

主矩:向C点简化

直接用C点的合力P代表分布力



向C点简化的主矢为平面平行力系的合力,作用在C点的合力P对刚体任意点的力矩等于分力的力矩之和

$$M_{C} = \int_{0}^{\frac{2}{3}l} q' \cdot \left(\frac{2}{3}l - x\right) dx - \int_{\frac{2}{3}l}^{l} q' \cdot \left(x - \frac{2}{3}l\right) dx$$

$$= \int_{0}^{\frac{2}{3}l} \frac{qx}{l} \cdot \left(\frac{2}{3}l - x\right) dx - \int_{\frac{2}{3}l}^{l} \frac{qx}{l} \cdot \left(x - \frac{2}{3}l\right) dx$$

$$= \frac{ql^{2}}{3} - \frac{ql^{2}}{3} = 0$$







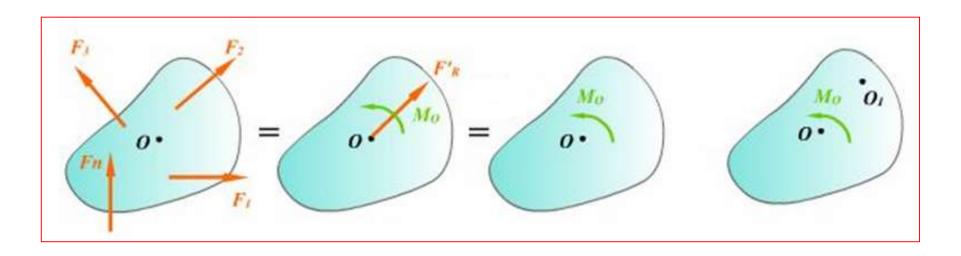






$$\overline{F}'_{R} = 0$$
  $M_{O} \neq 0$  合力偶

与简化中心的位置无关



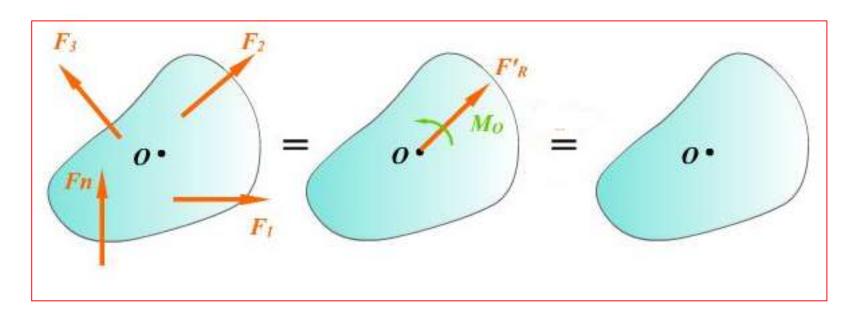
若以 $O_1$ 点简化,如何?

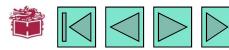
不变。力的平移定理(零力)



$$\overline{F}'_{R} = 0$$
  $M_{O} = 0$  平衡

与简化中心的位置无关平衡的刚体可以取任意点列平衡方程





## 某个平面任意力系向A与B两点简化的主矩均为零 ,此力系简化的最终结果可能是

- 一个非零力
- 一个非零力偶
- 平衡状态
- 以上均可能













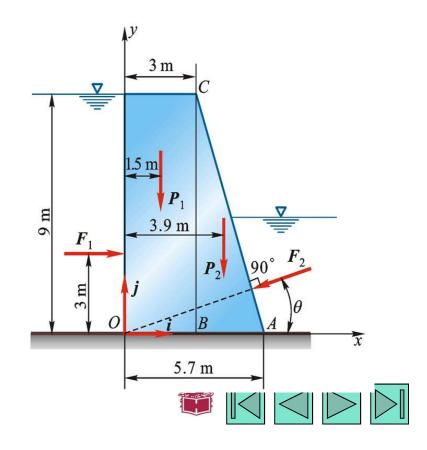
### 例2-10(平面力系的简化)

已知:  $P_1 = 450 \text{kN}, P_2 = 200 \text{kN}, F_1 = 300 \text{kN}, F_2 = 70 \text{kN}$ 

求: 力系向 O 点的简化结果;

合力与OA的交点到点O的距离x;

合力作用线方程。



### 解:

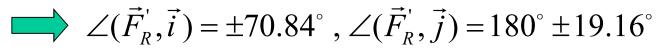
### (1) 向O点简化

主矢: 
$$\sum F_x = F_1 - F_2 \cos \theta = 232.9 \text{kN}$$
  
 $\sum F_y = -P_1 - P_2 - F_2 \sin \theta = -670.1 \text{kN}$ 

$$F_R' = \sqrt{(\sum F_x)^2 + (\sum F_y)^2} = 709.4 \text{kN}$$

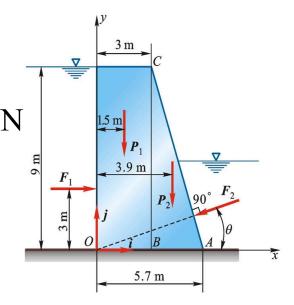
$$\cos(\vec{F}_{R}', \vec{i}) = \frac{\sum F_{x}}{F_{R}'} = 0.3283,$$

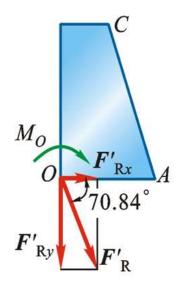
$$\cos(\vec{F}_{R}', \vec{j}) = \frac{\sum F_{y}}{F_{R}'} = -0.9446$$



#### 主矩:

$$M_O = \sum M_O(\vec{F})$$
  
=  $-3F_1 - 1.5P_1 - 3.9P_2 = -2355$ kN·m











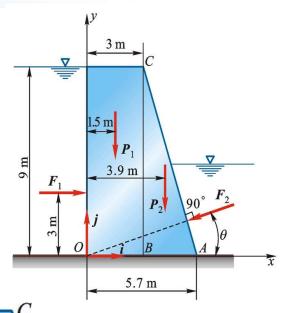


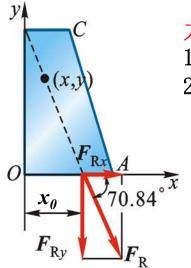


### (2) 合力与OA的交点到点O的距离x;

$$M_O = M_O(F_R) = M_O(F_{Rx}) + M_O(F_{Ry}) = 0 + F_{Ry}x$$

$$x_0 = \frac{M_O(F_R)}{F_{Ry}} = \frac{2355 \text{kN} \cdot \text{m}}{670.1 \text{kN}} = 3.514 \text{m}$$



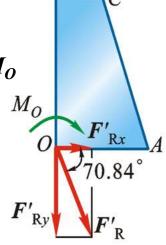


#### 力的作用点:

1)把力偶 $M_o$ 画成一对力

2)移动后 $F_R$ 对O点产生 $M_O$ 











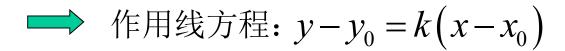




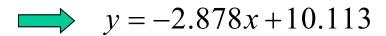
(3) 求合力作用线方程:

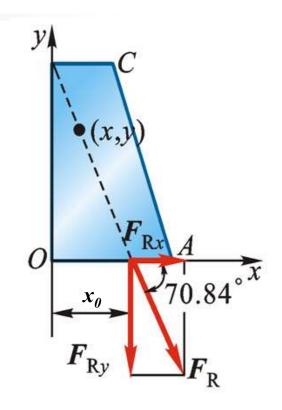
合力作用点为 $x_0$ =3.514m,  $y_0$ =0

合力与x轴夹角为:  $\angle(\vec{F}_{R},\vec{i}) = -70.84^{\circ}$ 



$$k = \tan(-70.84^{\circ}) = -2.878$$



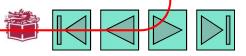


### 合力矩定理: 合力的力矩=分力的力矩和

$$M_{O} = \sum M_{O}(\vec{F}_{R}) = x \cdot F_{Ry} - y \cdot F_{Rx} = x \cdot F_{Ry}' - y \cdot F_{Rx}'$$

$$-2355 = x(-670.1) - y(232.9)$$

方法二





### 一. 平面任意力系的平衡方程

平面任意力系平衡的充要条件是:

$$\overline{F}'_{\rm R} = 0$$

力系的主矢和对任意点的主矩都等于零

$$M_O = 0$$

因为 
$$F'_{R} = \sqrt{(\sum F_{x})^{2} + (\sum F_{y})^{2}}$$
  $M_{O} = \sum M_{O}(\overline{F}_{i})$ 

平面任意力系的平衡方程(分量形式):

$$\begin{cases} \sum F_{x} = 0 \\ \sum F_{y} = 0 \\ \sum M_{O} = 0 \end{cases}$$

平面任意力系平衡的解析条件是: 所有各力在两个任选的坐标轴上的投 影的代数和分别等于零,以及各力对 于任意一点的矩的代数和也等于零.



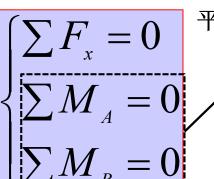








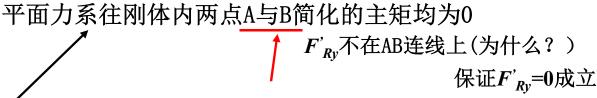
### 平面任意力系的平衡方程另两种形式



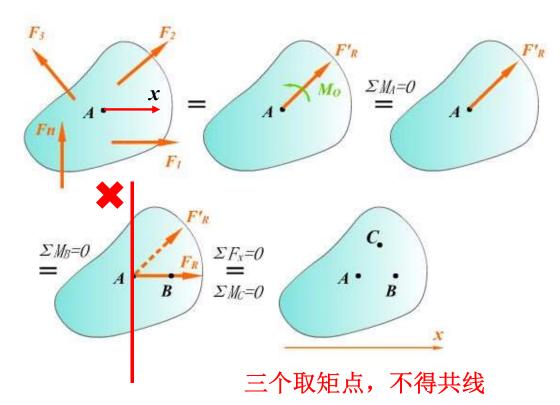
二矩式

$$\begin{cases} \sum M_{A} = 0 \\ \sum M_{B} = 0 \\ \sum M_{C} = 0 \end{cases}$$

三矩式



两矩点AB连线,不得与x轴垂直







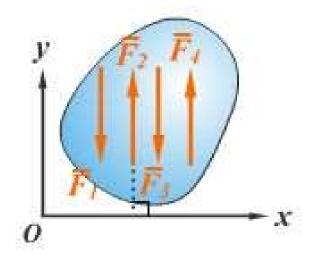


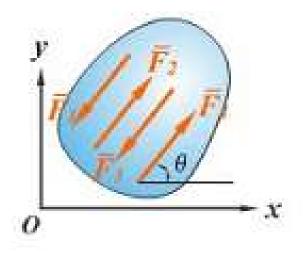






### 二. 平面平行力系的平衡方程





$$\sum F_{x} = 0$$
  $0 + 0 + 0 + \cdots = 0$ 

$$\sum F_{x} = 0 \quad F_{1} \cos \theta - F_{2} \cos \theta + F_{3} \cos \theta + \dots = 0$$

$$\sum F_{y} = 0 \quad F_{1} \sin \theta - F_{2} \sin \theta + F_{3} \sin \theta + \dots = 0$$

$$\begin{bmatrix} \sum F_y = 0 \\ \sum M_A = 0 \end{bmatrix}$$
 有一方向力平  
衡自然满足  
两个方程即可

$$\sum M_A = 0$$
 两点连线不得  $\sum M_B = 0$  与各力平行







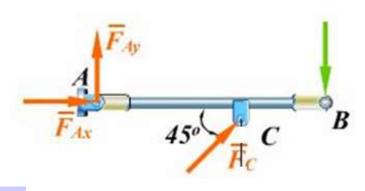


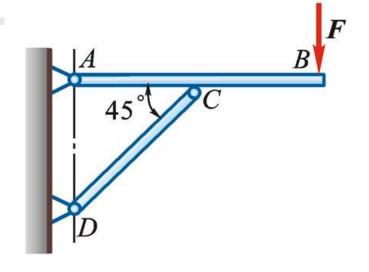


### 例2-11 (直接用平衡条件)

已知: AC = CB = l, F = 10 kN

求: 铰链A和DC杆受力.



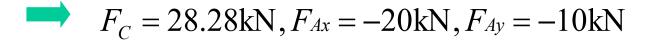


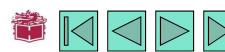
### 解: 取 AB梁, 画受力图.

$$\sum F_x = 0 \qquad F_{Ax} + F_C \cos 45^\circ = 0$$

$$\sum F_{v} = 0$$
  $F_{Av} + F_{C} \sin 45^{\circ} - F = 0$ 

$$\sum M_A = 0 \quad F_C \cos 45^\circ \cdot l - F \cdot 2l = 0$$





### 例2-12(止推轴承约束)

已知:  $P_1 = 10$ kN,  $P_2 = 40$ kN,尺寸如图。

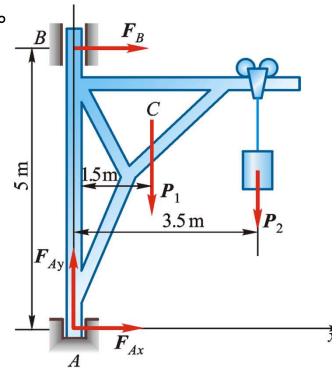
求: 轴承 A,B 处的约束力.

解: 取起重机,画受力图.

$$\sum F_{x} = 0 \qquad F_{Ax} + F_{B} = 0$$

$$\sum F_{v} = 0$$
  $F_{Ay} - P_1 - P_2 = 0$ 

$$\sum M_A = 0 \qquad -F_B \cdot 5 - 1.5 \cdot P_1 - 3.5 \cdot P_2 = 0$$





$$F_{Ay} = 50 \text{kN}$$
  $F_{B} = -31 \text{kN}$   $F_{Ax} = 31 \text{kN}$ 

$$F_{Ax} = 31$$
kN









### 例2-13 (分布力)

已知: P,q,a,M=qa.

求: 支座 A,B 处的约束力.

解: 取AB梁, 画受力图.

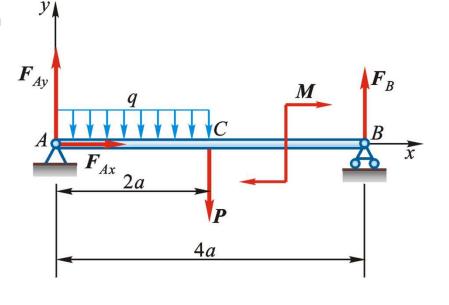
$$\sum F_{x} = 0$$
  $F_{Ax} = 0$ 

$$\sum M_A = 0 \quad F_B \cdot 4a - M - P \cdot 2a - q \cdot 2a \cdot a = 0$$

$$F_{B} = \frac{3}{4}P + \frac{1}{2}qa$$

$$\sum F_{y} = 0 \qquad F_{Ay} - q \cdot 2a - P + F_{B} = 0$$

$$F_{Ay} = \frac{P}{A} + \frac{3}{2}qa$$



### 可以选B计算吗?

一般选取最多未 知力的点列力矩 平衡方程(A点)



### 例2-14(固定端约束)

已知:  $P = 100 \text{kN}, M = 20 \text{kN} \cdot \text{m}$ ,

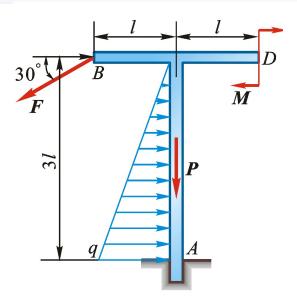
 $q = 20 \,\text{kN/m}, F = 400 \,\text{kN}, l = 1 \,\text{m}$ 

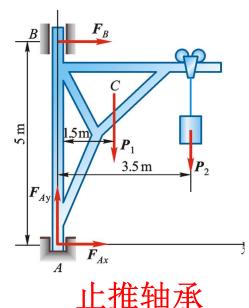
求: 固定端 A 处约束力.

A处的约束为[填空1]:

产生的约束力(力偶)共有[填空2]个;

系统受到[填空3]个主动力(力偶)作用。





作答









### 例2-14 (固定端约束)

已知:  $P = 100 \text{kN}, M = 20 \text{kN} \cdot \text{m},$ 

$$q = 20 \,\mathrm{kN/m}$$
,  $F = 400 \,\mathrm{kN}$ ,  $l = 1 \,\mathrm{m}$ 

求: 固定端 A 处约束力.

解:取T型刚架,画受力图.

其中 
$$F_1 = \frac{1}{2}q \times 3l = 30$$
kN

$$\sum F_x = 0$$
  $F_{Ax} + F_1 - F \sin 60^\circ = 0$ 

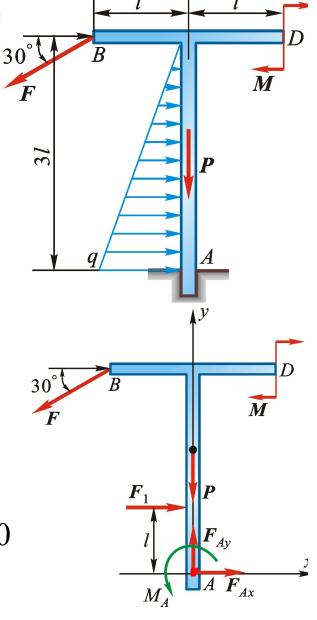
$$\sum F_{v} = 0 \quad F_{Ay} - P - F \cos 60^{\circ} = 0$$

$$\sum M_A = 0$$

$$M_A - M - F_1 \cdot l + F \cos 60^{\circ} \cdot l + F \sin 60^{\circ} \cdot 3l = 0$$



$$M_A = -1188$$
kN·m













#### 例2-15(翻倒问题)

已知:  $P_1 = 700 \text{kN}, P_2 = 200 \text{kN}, AB = 4 \text{m}$ 

求: (1)起重机满载和空载时不翻倒,平衡载重 $P_3$ ;

(2)  $P_3 = 180$ kN,轨道 AB给起重机轮子的约束力。

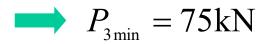
解: 取起重机,画受力图.

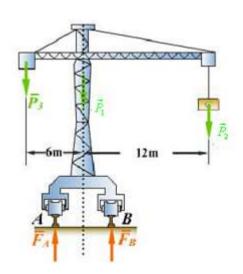
满载时,  $\vec{F}_A = 0$ ,

为不安全状况

$$\sum M_B = 0$$

$$P_{3\min} \cdot 8 + 2P_1 - 10P_2 = 0$$















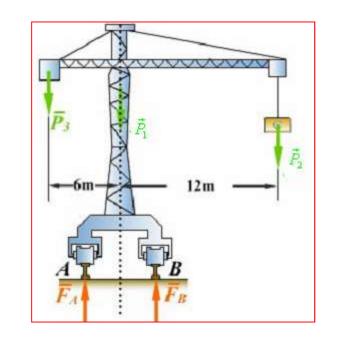
# 空载时, $\vec{F}_B = 0$ , 为不安全状况

$$\sum M_A = 0$$
  $4P_{3 \text{max}} - 2P_1 = 0$ 

$$F_{3 \text{max}} = 350 \text{kN}$$

 $75kN \le P_3 \le 350kN$ 

 $P_3 = 180$ kN 时



$$\sum M_A = 0 \quad 4P_3 - 2P_1 - 14P_2 + 4F_B = 0$$

$$\sum F_{iy} = 0 \quad F_A + F_B - P_1 - P_2 - P_3 = 0$$

$$F_{A} = 210 \text{kN}$$
  $F_{B} = 870 \text{kN}$ 











作业

教材习题: 2-21, 2-37, 2-39







