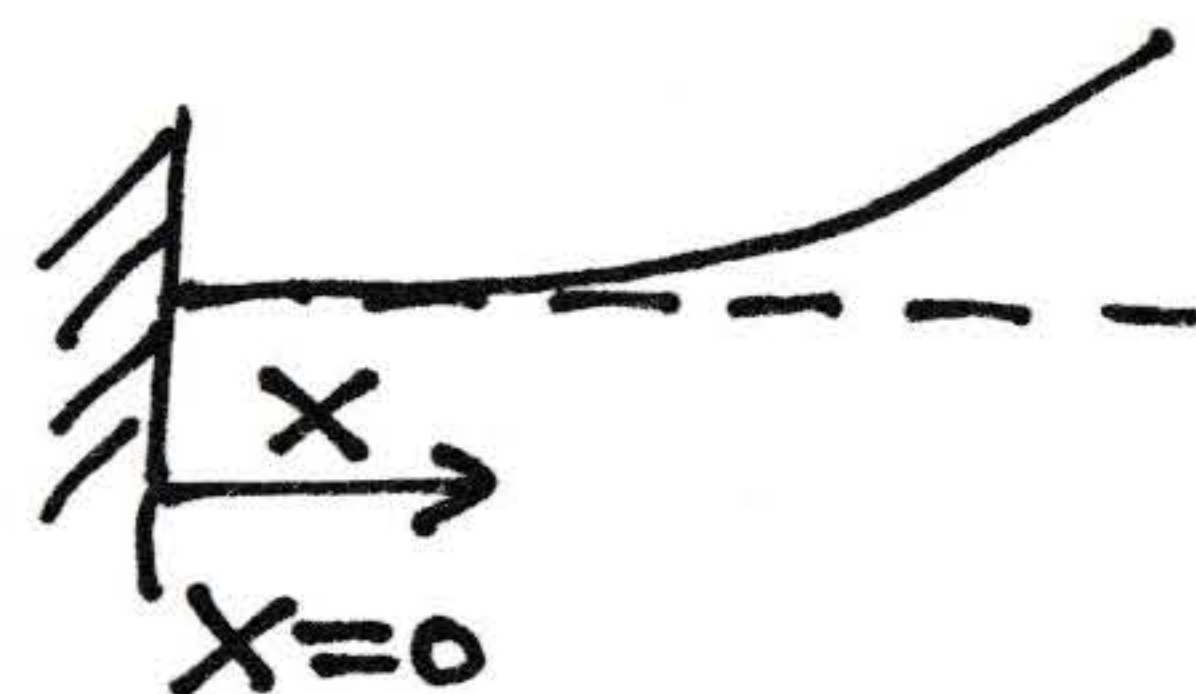


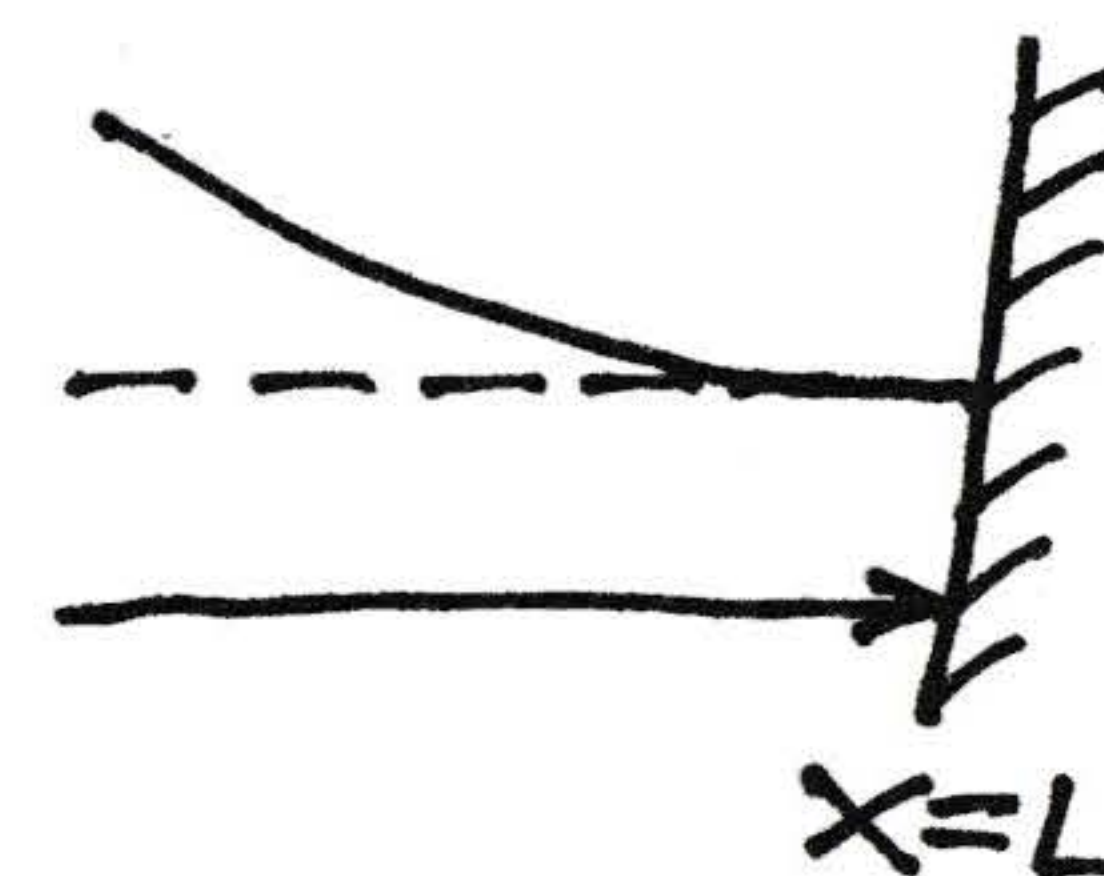
Simple BCs

1) Clamped BC:



$$V(0,t) = 0$$

$$\frac{\partial V}{\partial x}(0,t) = 0$$



$$V(L,t) = 0$$

$$\frac{\partial V}{\partial x}(L,t) = 0$$

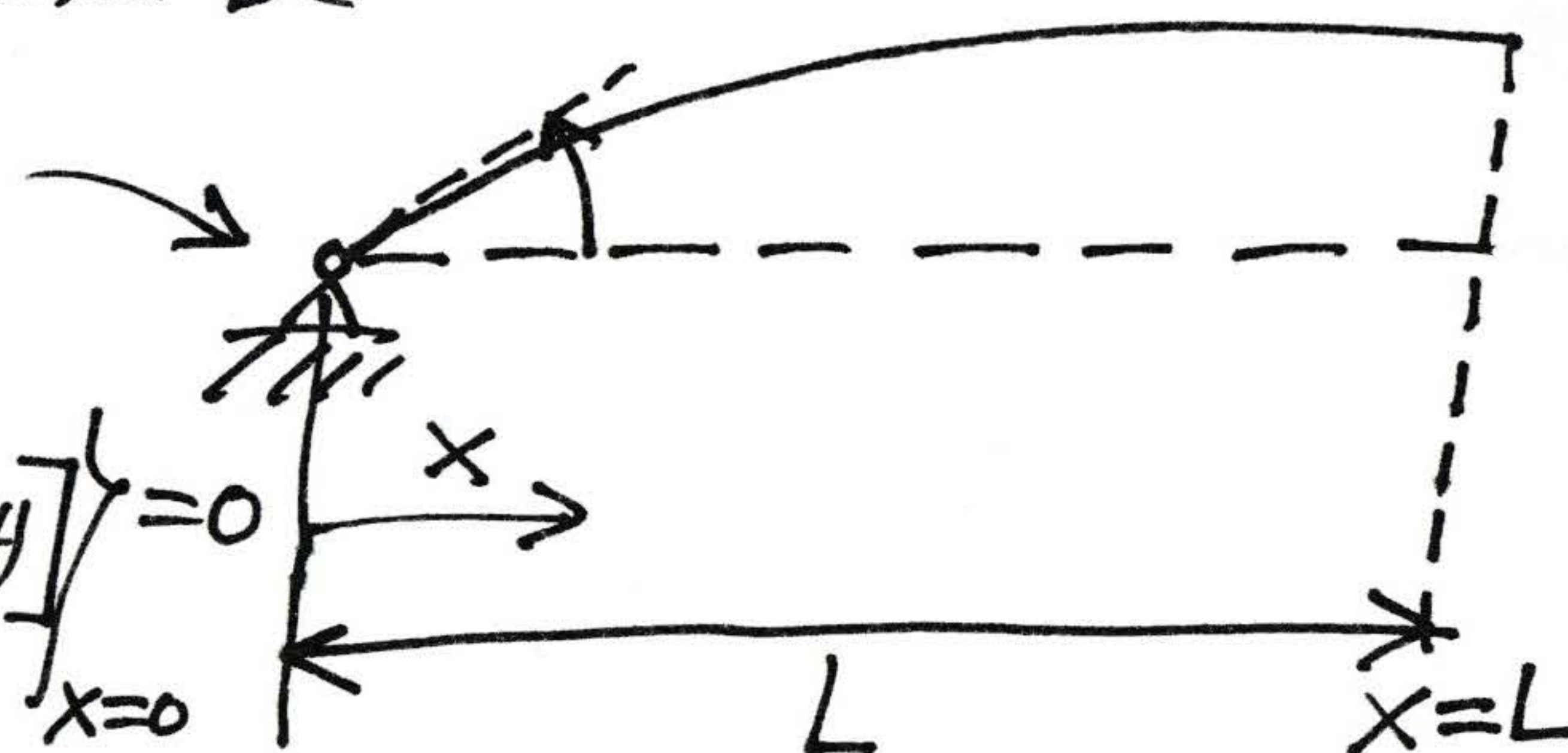
2) Free BC:

Simply-supported BC

$$V(0,t) = 0$$

$$Q(0,t) = 0 \Rightarrow$$

$$\Rightarrow \left\{ -\frac{\partial}{\partial x} \left[EI(x) \frac{\partial^2 V}{\partial x^2}(x,t) \right] \right\}_{x=0} = 0$$

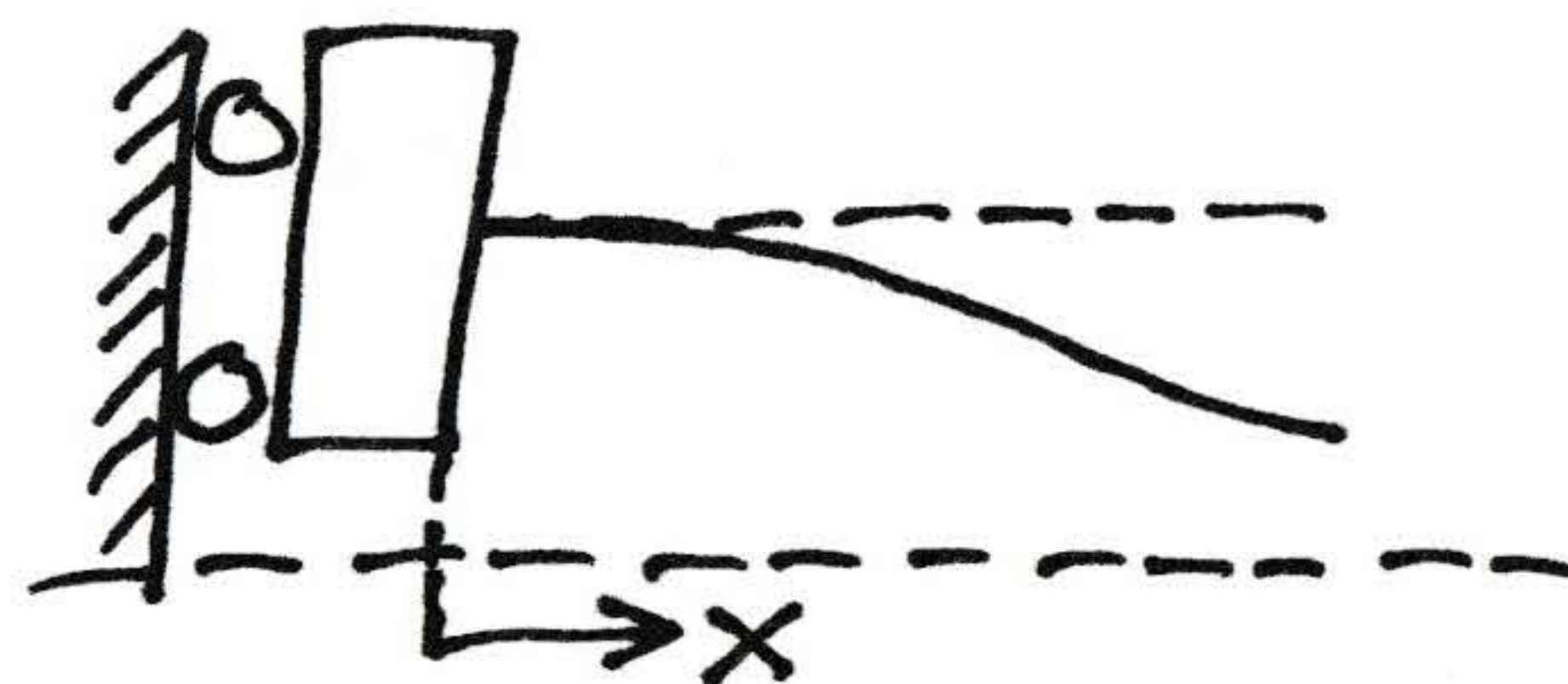


free BC

$$M(L,t) = 0 \Rightarrow \left\{ EI(x) \frac{\partial^2 V}{\partial x^2}(x,t) \right\}_{x=L} = 0 \Rightarrow \text{If } EI(L) \neq 0 \Rightarrow \frac{\partial^2 V}{\partial x^2}(L,t) = 0$$

$$Q(L,t) = 0 \Rightarrow \left\{ -\frac{\partial}{\partial x} \left[EI(x) \frac{\partial^2 V}{\partial x^2}(x,t) \right] \right\}_{x=L} = 0 \Rightarrow \text{If } EI(L) \neq 0 \Rightarrow \frac{\partial^3 V}{\partial x^3}(L,t) = 0$$

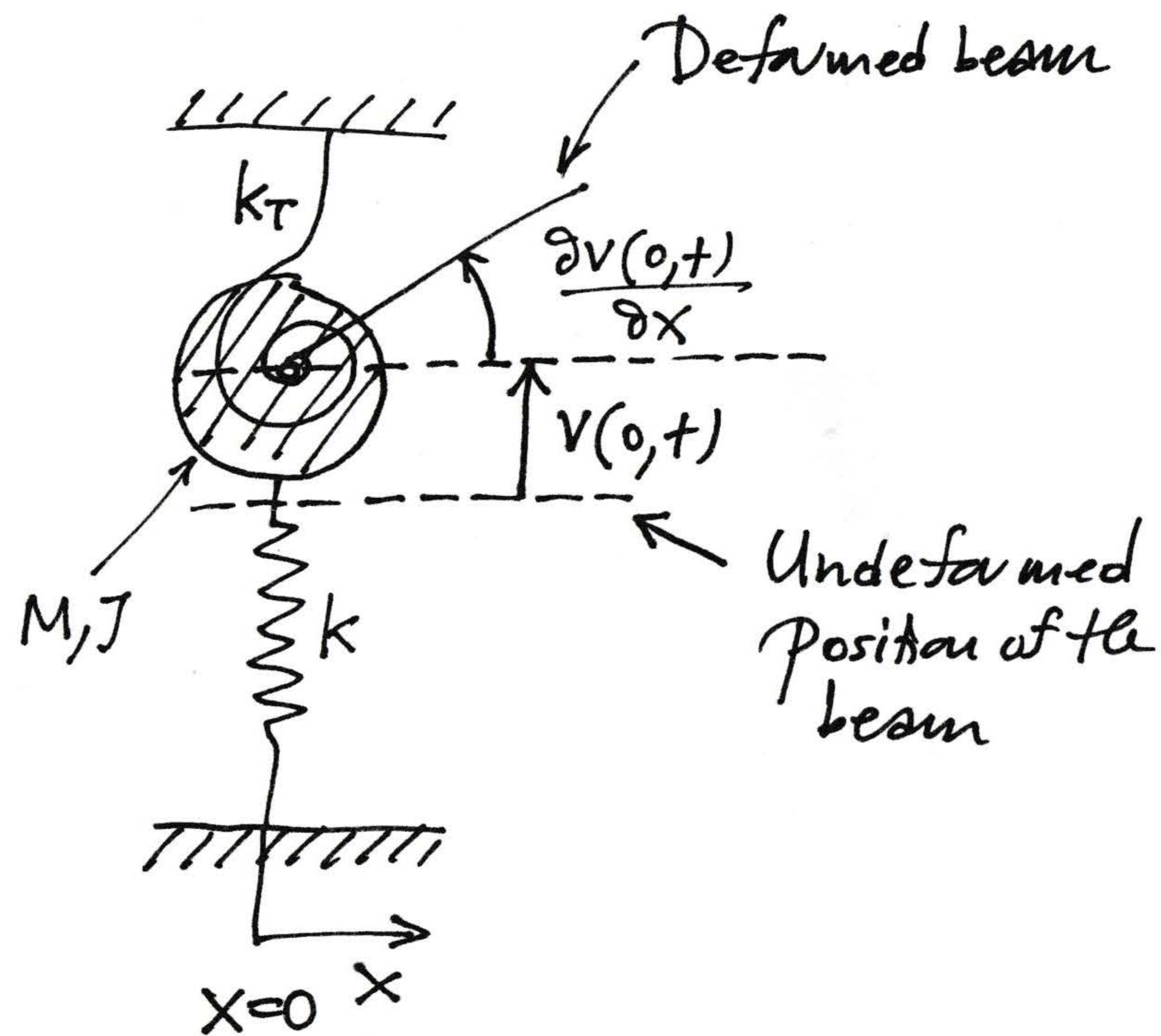
3) Semi-clamped BC:



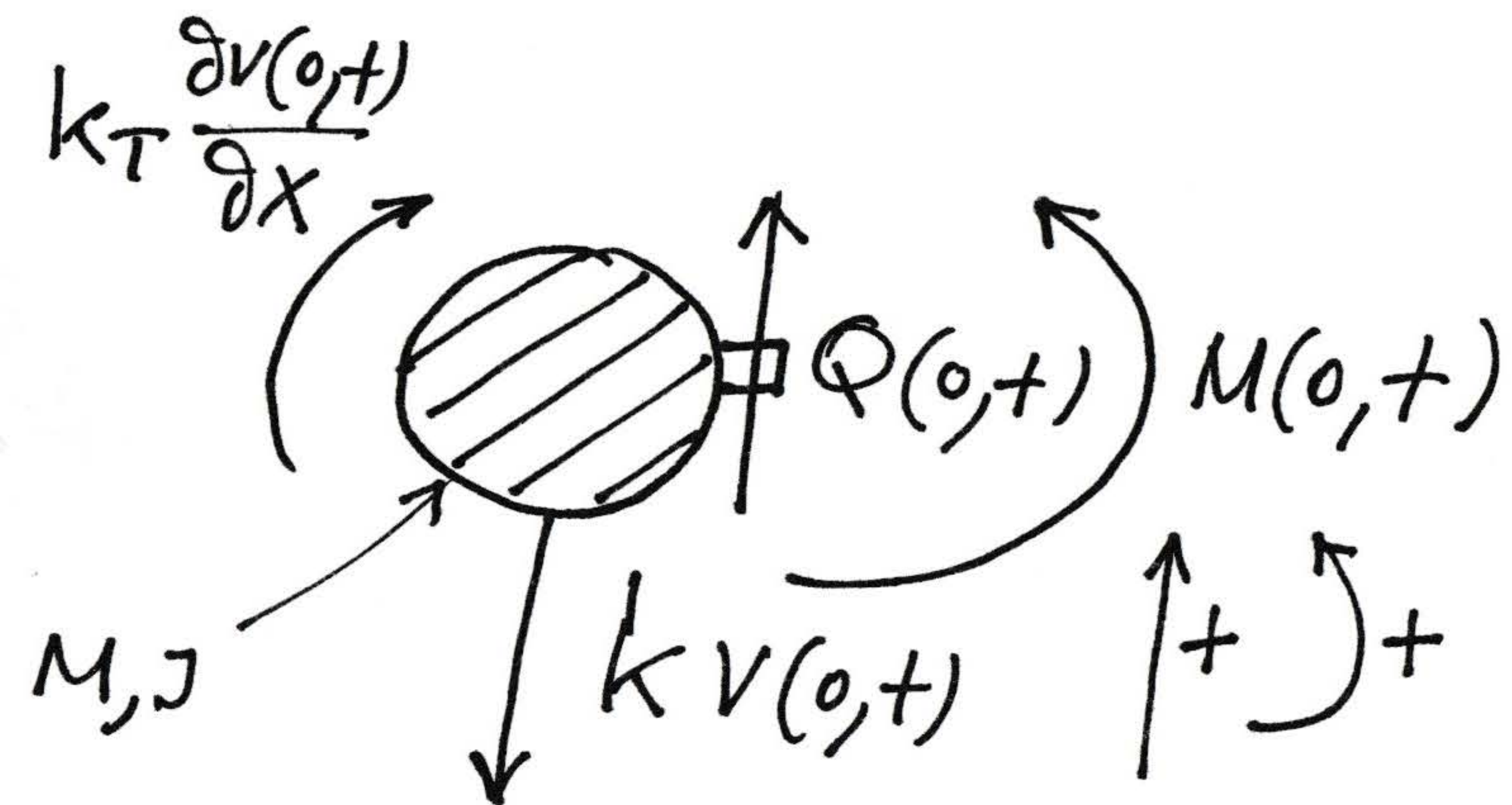
$$\frac{\partial V}{\partial x}(0,t) = 0$$

$$Q(0,t) = 0 \Rightarrow \left\{ -\frac{\partial}{\partial x} \left[EI(x) \frac{\partial^2 V}{\partial x^2}(x,t) \right] \right\}_{x=0} = 0$$

Non-simple BCs



Perform balance of moments and forces using a free-body diagram:



Perform balance of vertical forces \Rightarrow

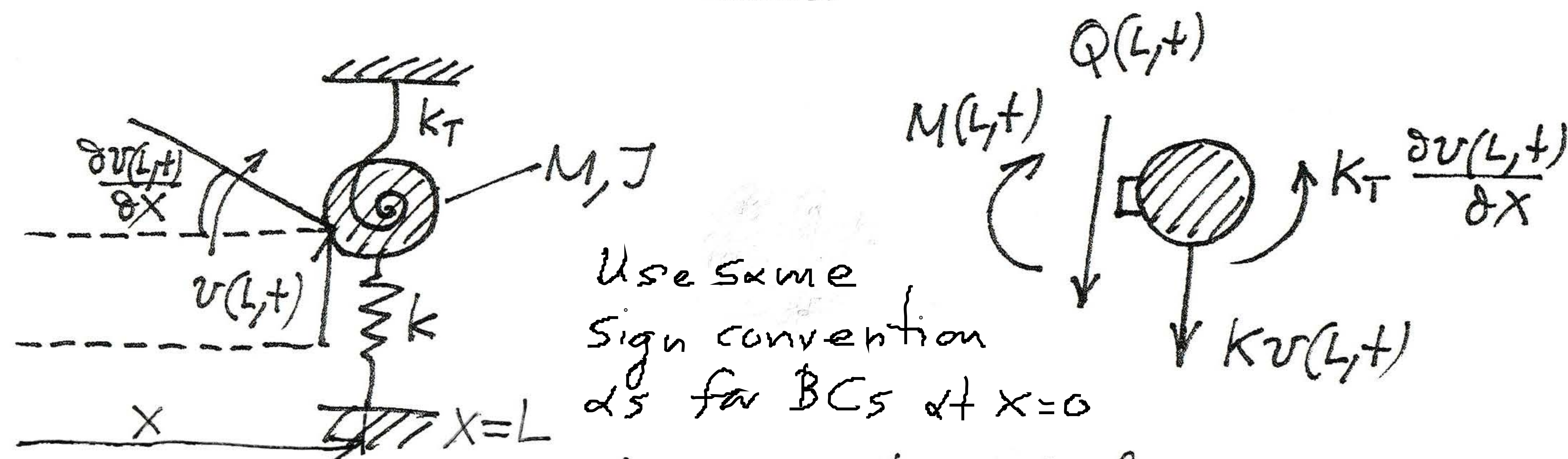
$$\Rightarrow M \frac{\partial^2 v(0,t)}{\partial t^2} = Q(0,t) - k v(0,t) \Rightarrow M \frac{\partial^2 v(0,t)}{\partial t^2} = \left\{ -\frac{\partial}{\partial x} \left[EI(x) \frac{\partial^2 v(x,t)}{\partial x^2} \right] \right\}_{x=0} - k v(0,t)$$

$$\Rightarrow \boxed{M \frac{\partial^2 v(0,t)}{\partial t^2} + \frac{\partial}{\partial x} \left[EI(0) \frac{\partial^2 v(0,t)}{\partial x^2} \right] + k v(0,t) = 0} \quad \text{BC at } x=0+$$

performing also balance of moments,

$$J \frac{\partial^2}{\partial t^2} \left[\frac{\partial v(0,t)}{\partial x} \right] = M(0,t) - k_T \frac{\partial v(0,t)}{\partial x} \Rightarrow$$

$$\Rightarrow \boxed{J \frac{\partial^3 v(0,t)}{\partial x \partial t^2} = EI(0) \frac{\partial^2 v(0,t)}{\partial x^2} - k_T \frac{\partial v(0,t)}{\partial x}} \quad \text{BC at } x=0+$$



performing balance of forces and moments before

$$\boxed{\frac{\partial}{\partial x} \left[EI(L) \frac{\partial^2 v(L,t)}{\partial x^2} \right] - k v(L,t) - M \frac{\partial^2 v(L,t)}{\partial t^2} = 0}$$

$$EI(L) \frac{\partial^2 v(L,t)}{\partial x^2} + k_T \frac{\partial v(L,t)}{\partial x} + J \frac{\partial^3 v(L,t)}{\partial x \partial t^2} = 0$$

BC at $x=L-$

The boundary value problem (I ignore the forcing at this stage)

Assume the general beam equation,

$$\frac{\partial^2}{\partial x^2} \left[EI(x) \frac{\partial^2 v(x,t)}{\partial x^2} \right] + m(x) \frac{\partial^2 v(x,t)}{\partial t^2} = 0 \quad (1)$$

$$0 \leq x \leq L, t \geq 0$$

Assume simple BCs, i.e., $u(0,t) = u(L,t) = 0$

$$\frac{\partial u}{\partial x}(0,t) = \frac{\partial u}{\partial x}(L,t) = 0 \quad (1a)$$



Initial conditions, $u(x,0) = g(x)$, $\frac{\partial u}{\partial t}(x,0) = h(x)$

Again, as for the case of the generalized wave equation we assume separation of space and time in the basic solution,

$$v(x,t) = f(t) \phi(x) \quad (2)$$

substituting back into the governing equation,

$$\frac{\partial^2}{\partial x^2} \left[EI(x) \frac{d^2 \phi(x)}{dx^2} f(t) \right] + m(x) \frac{d^2 f(t)}{dt^2} \phi(x) = 0 \Rightarrow$$

$$\Rightarrow f(t) \frac{d^2}{dx^2} \left[EI(x) \frac{d^2 \phi(x)}{dx^2} \right] + m(x) \phi(x) \frac{d^2 f(t)}{dt^2} = 0 \Rightarrow$$

$$\Rightarrow \frac{\ddot{f}(t)}{f(t)} = - \frac{\frac{d^2}{dx^2} \left[EI(x) \frac{d^2 \phi(x)}{dx^2} \right]}{m(x) \phi(x)} = -\omega^2 \quad (3)$$

$(\dot{}) \equiv \frac{d}{dt}$

Depends on t Depends on x

Then we get the following two sub problems in time and space:

$$\ddot{f}(t) + \omega^2 f(t) = 0, \quad t \geq 0 \Rightarrow f(t) = A_1 \cos \omega t + B_1 \sin \omega t \quad (4a)$$

$$\frac{d^2}{dx^2} \left[EI(x) \frac{d^2 \phi(x)}{dx^2} \right] - \omega^2 m(x) \phi(x) = 0, \quad 0 \leq x \leq L \quad (4b)$$

$$\phi(0) = \phi(L) = 0$$

$$\phi'(0) = \phi'(L) = 0$$

$()' \equiv \frac{d}{dx}$

For the general problem (4b), if we can find four linearly independent solutions, then the general solution is expressed by superposition as, $\phi(x) = C_1 \phi_1(x, \omega) + C_2 \phi_2(x, \omega) + C_3 \phi_3(x, \omega) + C_4 \phi_4(x, \omega)$ where $\phi_i(x)$, $i=1,2,3,4$ are the four linearly independent solutions of (4b)

To obtain an analytical solution we examine the uniform beam with $m(x)=m$, $EI(x)=EI \Rightarrow EI \frac{d^4 \phi(x)}{dx^4} - \omega^2 m \phi(x) = 0 \Rightarrow$

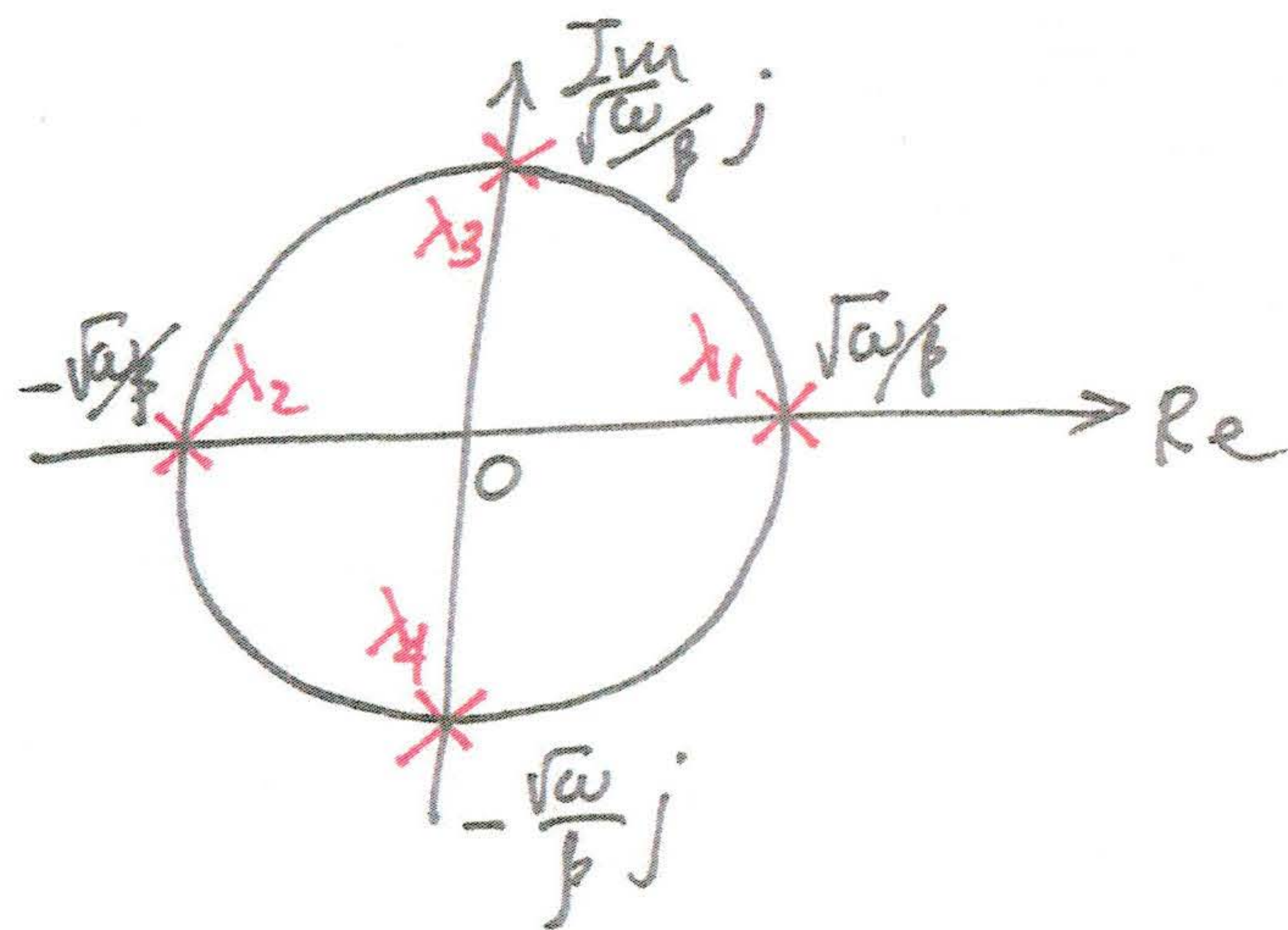
$$\Rightarrow \frac{d^4 \phi(x)}{dx^4} - \left(\frac{\omega^2}{\beta^4}\right) \phi(x) = 0, \quad \beta^4 = \frac{EI}{m}$$

The characteristic equation for this problem is $\lambda^4 - \frac{\omega^2}{\beta^4} = 0 \Rightarrow$

$$\Rightarrow \lambda^4 = \frac{\omega^2}{\beta^4} e^{j2n\pi}, \quad n=0,1,2,3, \quad j = (-1)^k \Rightarrow \lambda_i = \sqrt[4]{\frac{\omega^2}{\beta^4}} e^{j\frac{2in}{4}} =$$

$$= \frac{\sqrt{\omega}}{\beta} e^{j\frac{in}{2}}, \quad i=1,2,3 \Rightarrow$$

$$\Rightarrow \lambda_1 = \frac{\sqrt{\omega}}{\beta}, \quad \lambda_2 = -\frac{\sqrt{\omega}}{\beta}, \quad \left. \begin{array}{l} \lambda_3 = j\frac{\sqrt{\omega}}{\beta}, \quad \lambda_4 = -j\frac{\sqrt{\omega}}{\beta} \end{array} \right\} \Rightarrow$$



$$\Rightarrow \left. \begin{array}{l} \phi_1(x, \omega) = e^{\frac{\sqrt{\omega}}{\beta} x} \\ \phi_2(x, \omega) = e^{-\frac{\sqrt{\omega}}{\beta} x} \\ \phi_3(x, \omega) = e^{j\frac{\sqrt{\omega}}{\beta} x} \\ \phi_4(x, \omega) = e^{-j\frac{\sqrt{\omega}}{\beta} x} \end{array} \right\} \begin{array}{l} \text{Should take into} \\ \text{account that} \\ \sinh x = \frac{e^x - e^{-x}}{2} \\ \cosh x = \frac{e^x + e^{-x}}{2} \\ e^{j\theta} = \cos \theta + j \sin \theta \end{array}$$

Then, the general solution is

$$\varphi(x) = A_1 \varphi_1(x, \omega) + A_2 \varphi_2(x, \omega) + A_3 \varphi_3(x, \omega) + A_4 \varphi_4(x, \omega) \Rightarrow$$

$$\Rightarrow \varphi(x) = C_1 \cos \frac{\sqrt{\omega} x}{\beta} + C_2 \sin \frac{\sqrt{\omega} x}{\beta} + C_3 \cosh \frac{\sqrt{\omega} x}{\beta} + C_4 \sinh \frac{\sqrt{\omega} x}{\beta}$$

Note that $(\sinh x)' = \cosh x$, $(\cosh x)' = \sinh x$

$$\text{BCs: } \varphi(0) = \varphi(L) = \varphi'(0) = \varphi'(L) = 0 \Rightarrow$$

\Rightarrow Substituting into $\varphi(x)$ we get a condition

$$\det[A(\omega)] = 0 \Rightarrow \text{Compute the eigenvalues (nat. frequencies)}$$

and then the modes of the problem.

