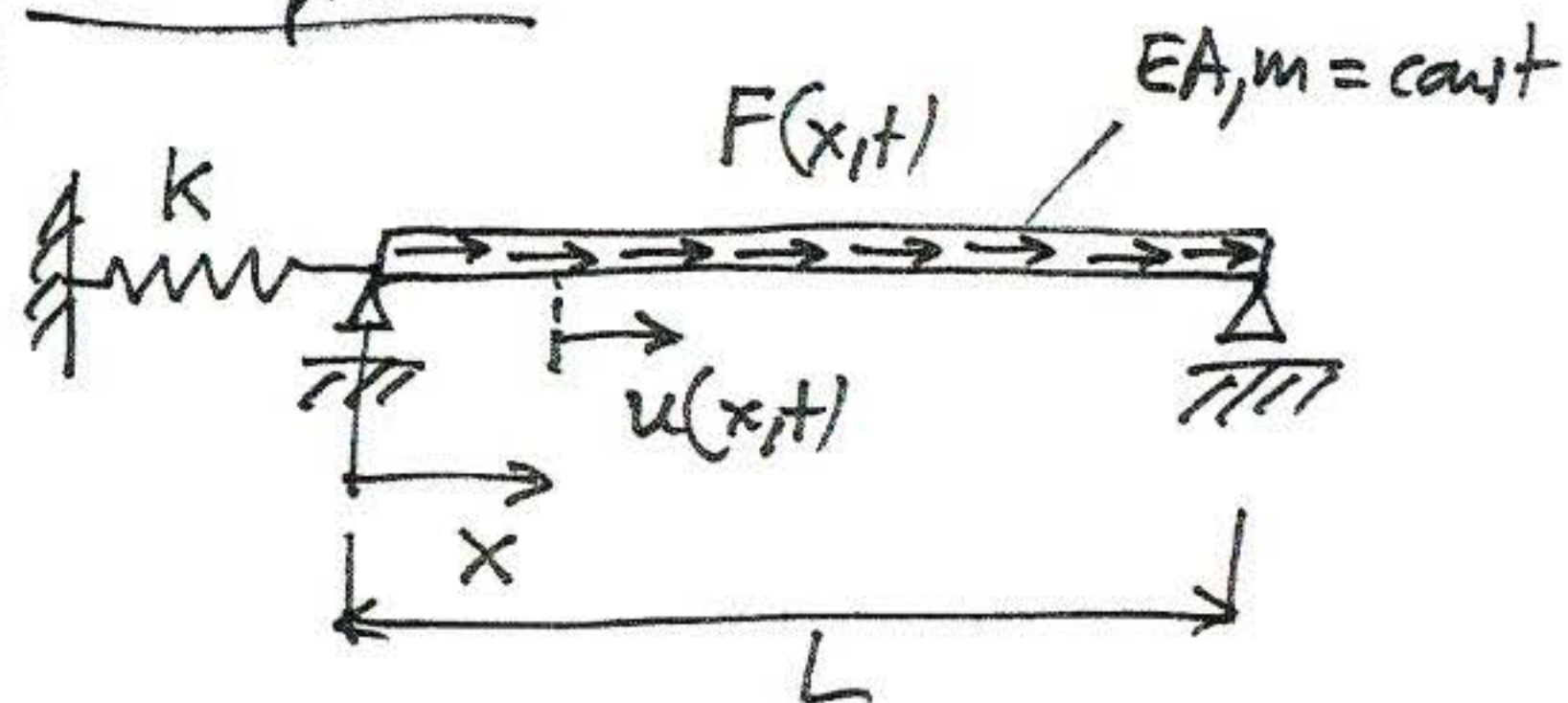


Example 3

$$EA \frac{\partial^2 u}{\partial x^2} + F(x,t) = m \frac{\partial^2 u}{\partial t^2}, \quad 0 \leq x \leq L \quad (1)$$

$$t \geq 0$$

$$\left. \begin{aligned} EA \frac{\partial u(0,t)}{\partial x} - k u(0,t) &= 0 \\ \frac{\partial u(L,t)}{\partial x} &= 0 \end{aligned} \right\} \text{BCs} \quad (1a)$$

$$u(x,0) = g(x), \quad \frac{\partial u}{\partial t}(x,0) = h(x) \quad (1b)$$

first we compute the modes of this system  $\Rightarrow \varphi''(x) + \left(\frac{\omega}{c}\right)^2 \varphi(x) = 0 \quad (2)$

$$\left. \begin{aligned} EA \varphi'(0) - k \varphi(0) &= 0 \\ \varphi'(L) &= 0 \end{aligned} \right\} \quad (2a)$$

Solving for  $\varphi(x) \Rightarrow \varphi(x) = C_1 \cos \frac{\omega x}{c} + C_2 \sin \frac{\omega x}{c} \Rightarrow$  Applying (2a)  $\Rightarrow$

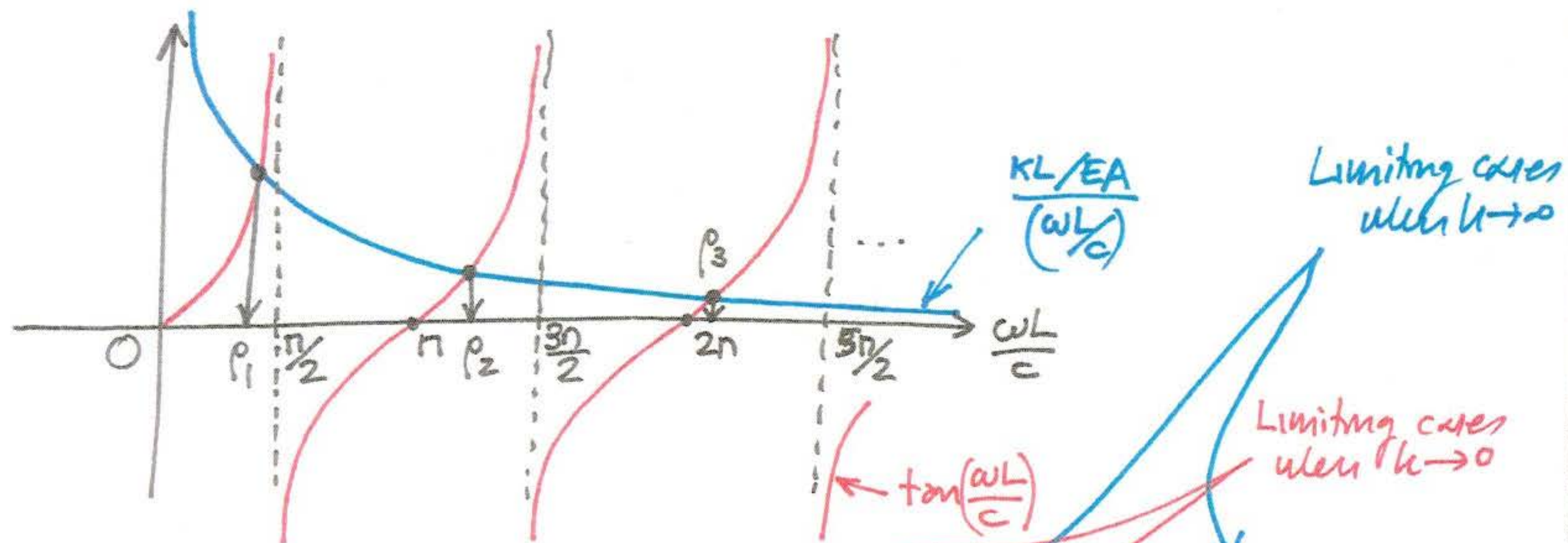
$$\Rightarrow \left. \begin{aligned} EA C_2 \frac{\omega}{c} - k C_1 &= 0 \\ -C_1 \frac{\omega}{c} \sin \frac{\omega L}{c} + C_2 \frac{\omega}{c} \cos \frac{\omega L}{c} &= 0 \end{aligned} \right\} \Rightarrow \begin{bmatrix} -k & EA \frac{\omega}{c} \\ -\sin \frac{\omega L}{c} & \cos \frac{\omega L}{c} \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \Rightarrow$$

$\Rightarrow$  for nontrivial solutions we require that  $\det[\quad] = 0 \Rightarrow$

$$\Rightarrow -k \cos \frac{\omega L}{c} + EA \frac{\omega}{c} \sin \frac{\omega L}{c} = 0 \Rightarrow \tan\left(\frac{\omega L}{c}\right) = \frac{k/EA}{\omega/c} = \frac{\frac{k}{EA}}{\left(\frac{\omega L}{c}\right) \frac{1}{L}} \Rightarrow \boxed{\tan\left(\frac{\omega L}{c}\right) = \frac{KL}{EA \left(\frac{\omega L}{c}\right)}}$$

Frequency equation





So we get a countable infinity of roots  $0 < \rho_1 < \pi/2, \pi < \rho_2 < 3\pi/2, \dots \Rightarrow$

$\Rightarrow$  Then we obtain the corresponding natural frequencies  $\omega_i L/c = \rho_i \Rightarrow$

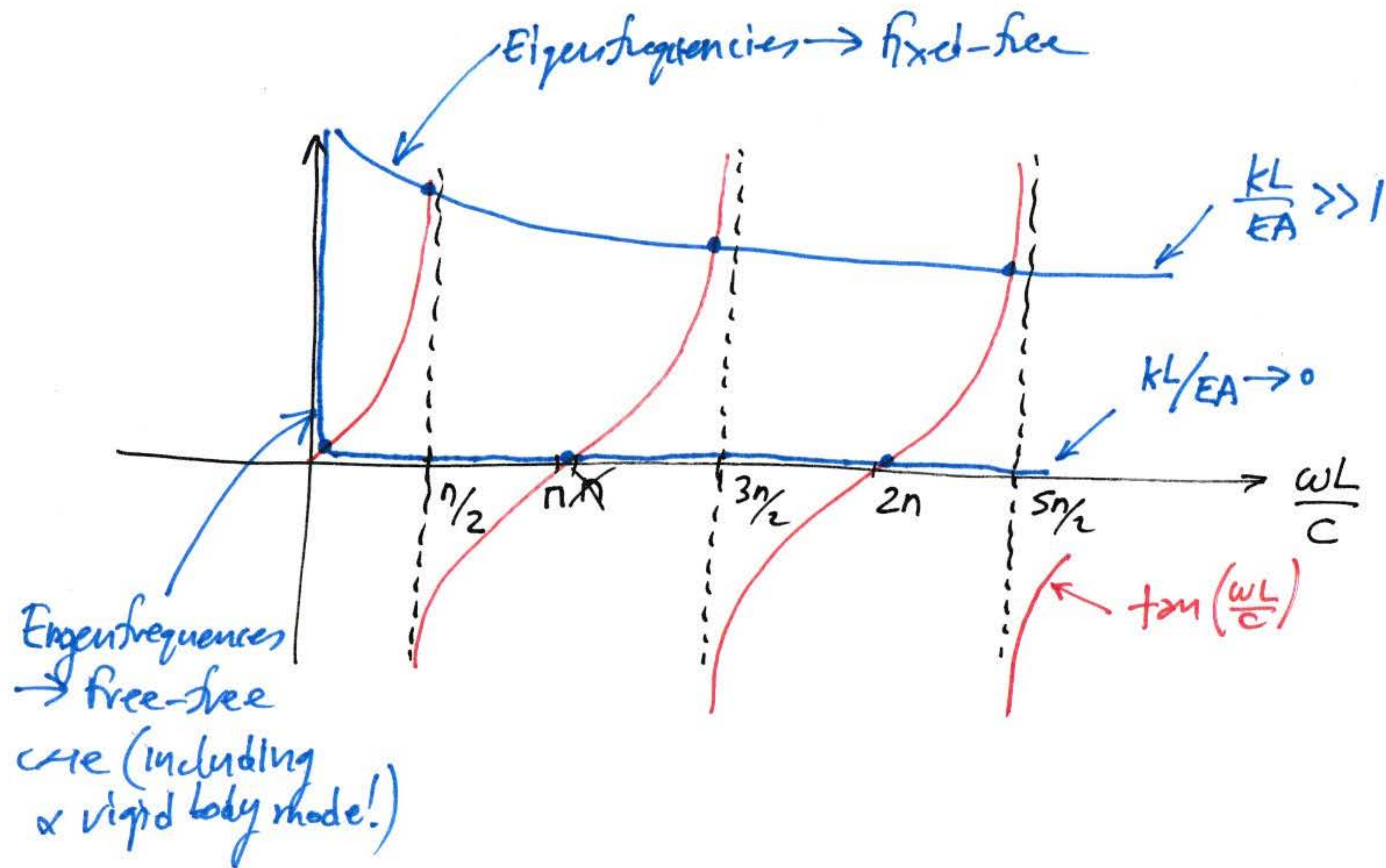
$\Rightarrow \omega_i = \frac{c \rho_i}{L}, i=1,2,\dots \Rightarrow$  Then we can compute the corresponding eigenfunction

by noting that  $C_2 = C_1 \tan \frac{\omega_i L}{c} \Rightarrow \varphi_i(x) = C_i \left( \cos \frac{\omega_i x}{c} + \tan \frac{\omega_i L}{c} \sin \frac{\omega_i x}{c} \right) \Rightarrow$

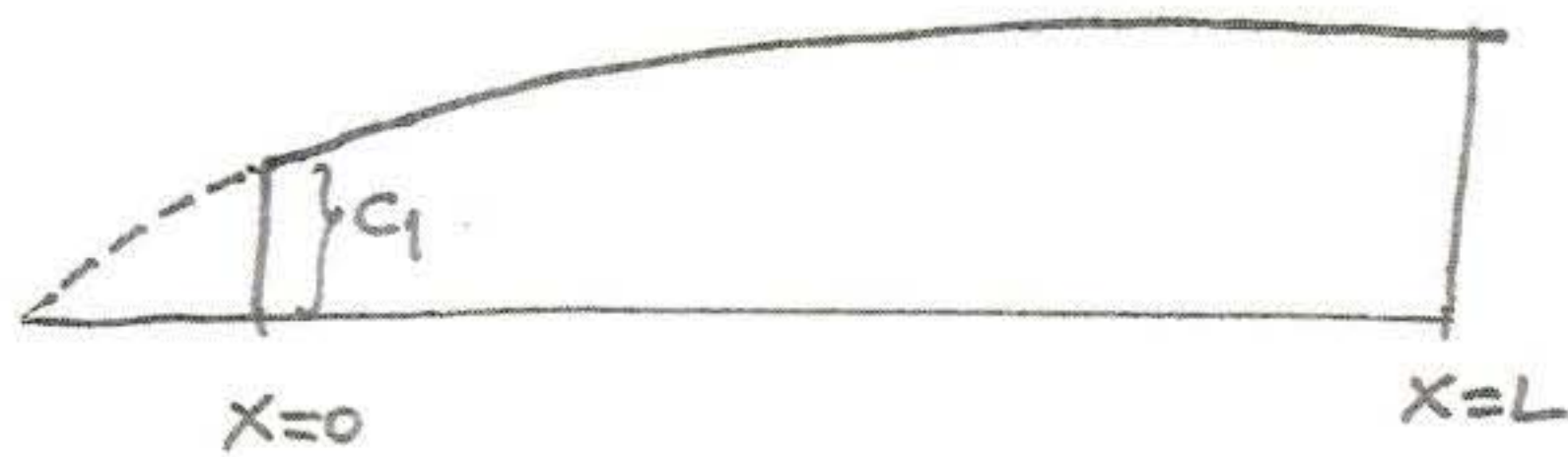
$\Rightarrow \varphi_i(x) = C_i \frac{\cos(\rho_i \frac{L-x}{L})}{\cos \rho_i}$  Eigenfunctions for  $i=1,2,\dots$  Of course, we may

orthonormalize the eigenfunctions  $\varphi_i(x)$  using the formulas for 'non-simple' BCs derived in earlier lecture.



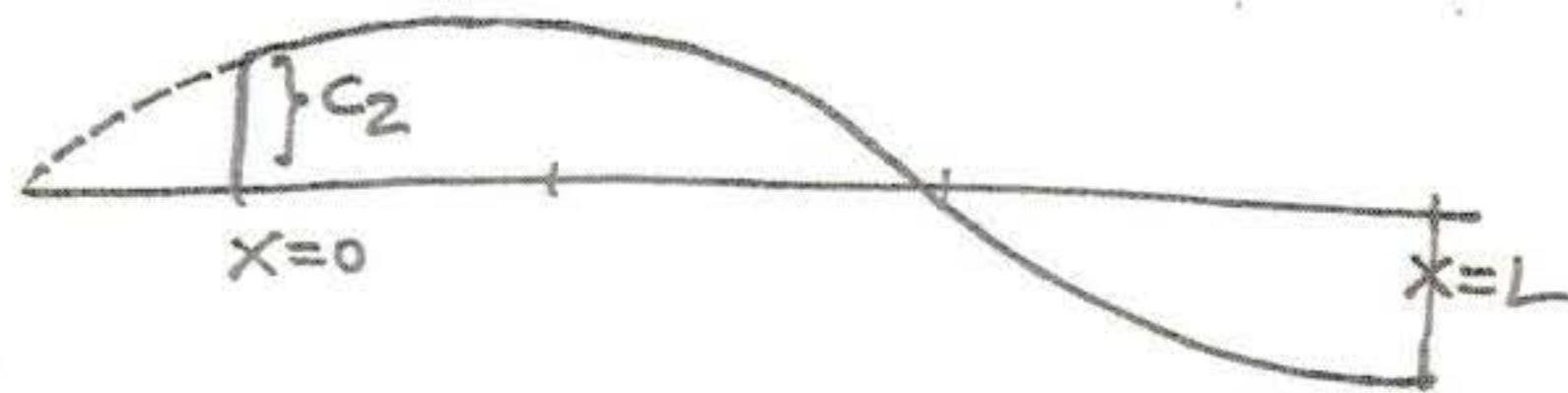





 $\phi_1(x)$ 

To perform orthonormalization we require that for each eigenfunction  $\phi_i(x)$  it holds that,

$$\int_0^L m \phi_i^2(x) dx = 1, i=1,2,\dots \Rightarrow$$


 $\phi_2(x)$ 

In simple form since the mass is uniform

$$\Rightarrow \int_0^L m c_i^2 \frac{\cos^2\left(\rho_i \frac{L-x}{L}\right)}{\cos^2 \rho_i} dx = 1 \Rightarrow$$


 $\phi_3(x)$ 
 $\vdots$ 

$$\Rightarrow c_i = 2 \cos \rho_i \sqrt{\frac{\rho_i}{Lm(2\rho_i + \sin 2\rho_i)}} \\ i=1,2,\dots$$

Then, the orthonormalized eigenfunctions are,

$$\phi_i(x) = 2 \sqrt{\frac{\rho_i}{Lm(2\rho_i + \sin 2\rho_i)}} \cos \rho_i \frac{L-x}{L}, \quad \omega_i = \rho_i \frac{c}{L}, \quad i=1,2,\dots$$

Note that the stiffness-orthogonality condition won't be simple due to the local stiffness at  $x=0$ .



Remark: What happens when  $k \rightarrow 0$ ? Suppose that  $\frac{KL}{EA} \ll 1 \Rightarrow$

$$\Rightarrow \tan\left(\frac{\omega L}{C}\right) = \frac{KL/EA}{\left(\frac{\omega L}{C}\right)} \Rightarrow \tan \frac{\omega L}{C} \ll 1 \Rightarrow \tan \frac{\omega L}{C} \sim \frac{\omega L}{C} \Rightarrow$$

For the first branch of  $\tan \nearrow$

$$\Rightarrow \frac{\omega_1 L}{C} \sim \frac{KL/EA}{\frac{\omega_1 L}{C}} \Rightarrow \left(\frac{\omega_1 L}{C}\right)^2 \sim \frac{KL}{EA} \Rightarrow \omega_1^2 \sim \frac{C^2}{L^2} \frac{KL}{EA} = \frac{EA}{mL} \frac{K}{EA} = \frac{K}{mL} \Rightarrow$$

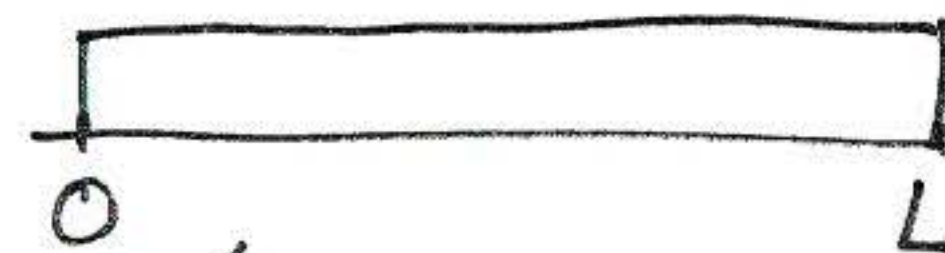
$$\Rightarrow \omega_1^2 \sim \frac{K}{mL}$$

then we get the approximation that the rod moves as a rigid body

Fixed 

So as  $K \rightarrow 0 \Rightarrow \omega_1^2 \rightarrow 0 \Rightarrow \phi_1(x) = \text{const}$

$M = mL$



Hence the first mode becomes the rigid-body mode.

But what happens to higher modes? As  $K \rightarrow 0 \Rightarrow \tan \frac{\omega L}{C} \rightarrow 0 \Rightarrow$

For other branches of  $\tan$

$$\Rightarrow \frac{\omega_i L}{C} \rightarrow \pi, 2\pi, 3\pi, \dots \quad (\text{the natural frequencies of free-free rod})$$

What happens when  $K \rightarrow \infty$ ? In that case,  $\phi_i \rightarrow$

Natural frequencies of fixed-free rod!

$$\frac{(2(i-1)+1)\pi}{2} = \frac{(2i-1)\pi}{2}, \quad i=1, 2, 3, \dots$$