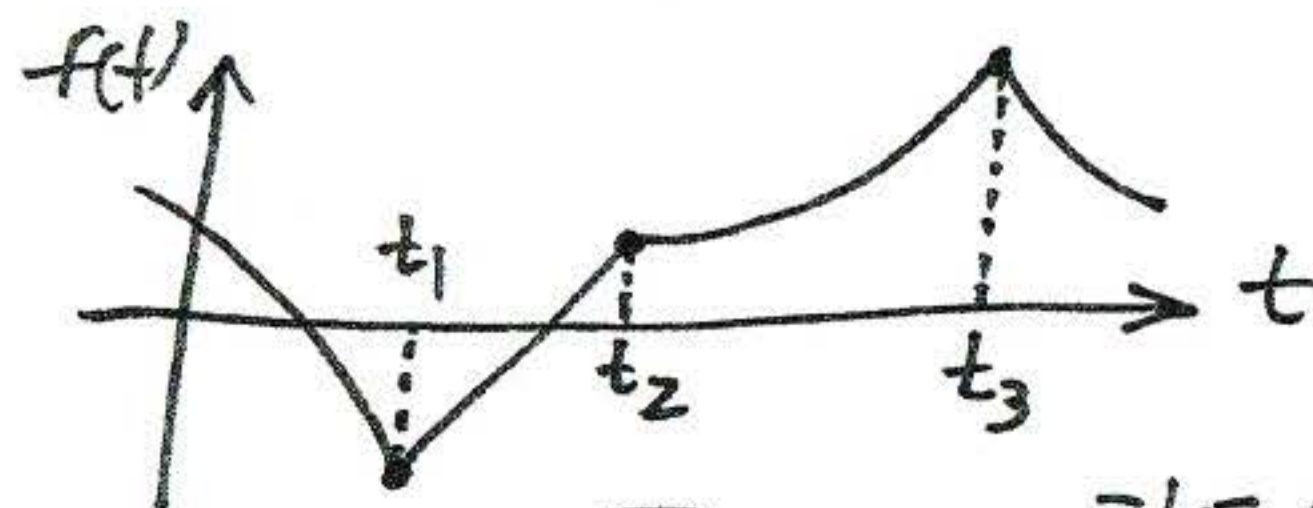


1.2 Forced Response

Aside: A function $f(t)$ is Laplace-transformable, if it is of Exponential order (EO) and is almost piecewise continuous (APC). A function $f(t)$ is of EO if $\exists \alpha \in \mathbb{R}$, such that: $\lim_{t \rightarrow \infty} f(t)e^{-\alpha t} = 0$ when $\alpha > \alpha_0$. A function $f(t)$ is APC in

every finite interval, if there are a finite number of points t_1, \dots, t_N such that at each of these points the behavior of $f(t)$ is described as follows:
 t_k is a singular point of $f(t)$, but in some neighborhood of t_k , $|t - t_k| < \delta$ the function has the property $|f(t)| < \frac{M}{(t - t_k)^\nu}$, $\nu < 1$, where M is a constant.
 Then $|f(t)|$ is integrable and $f(t)$ is absolutely integrable at each interval containing $t_k \Rightarrow$ An APC function $f(t)$ is integrable over any finite interval in t .

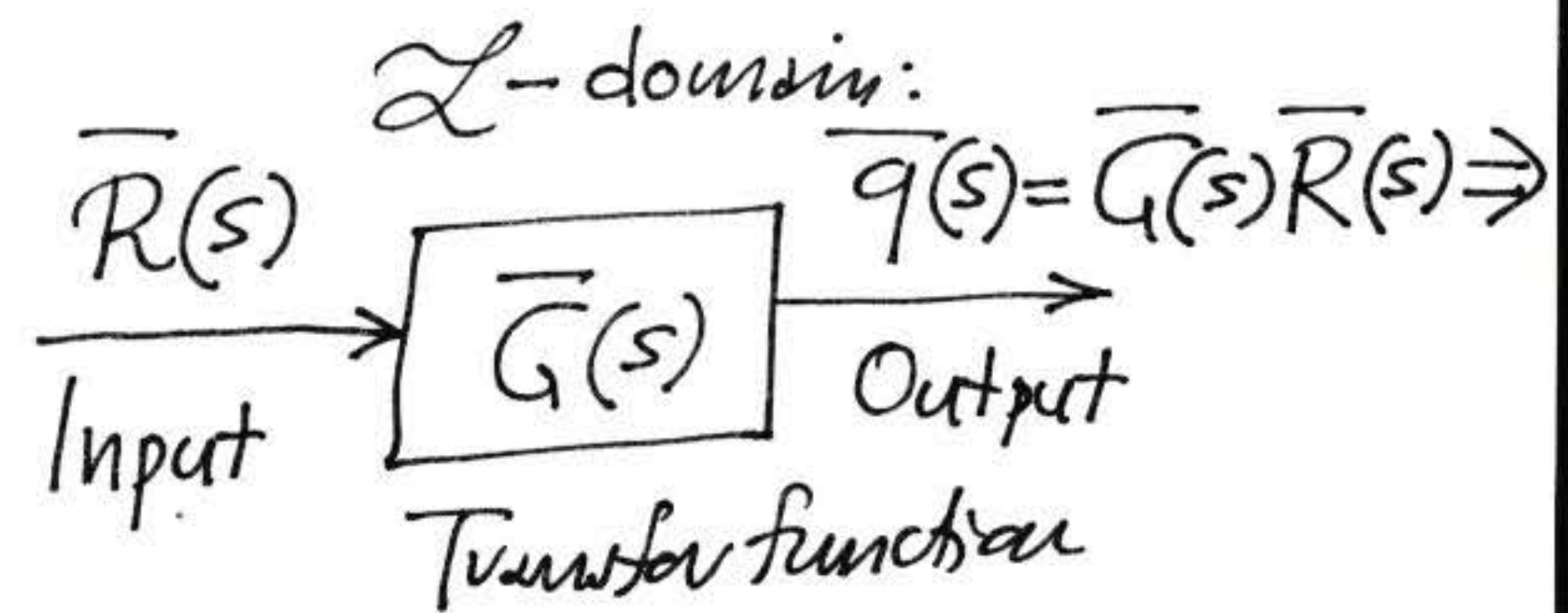
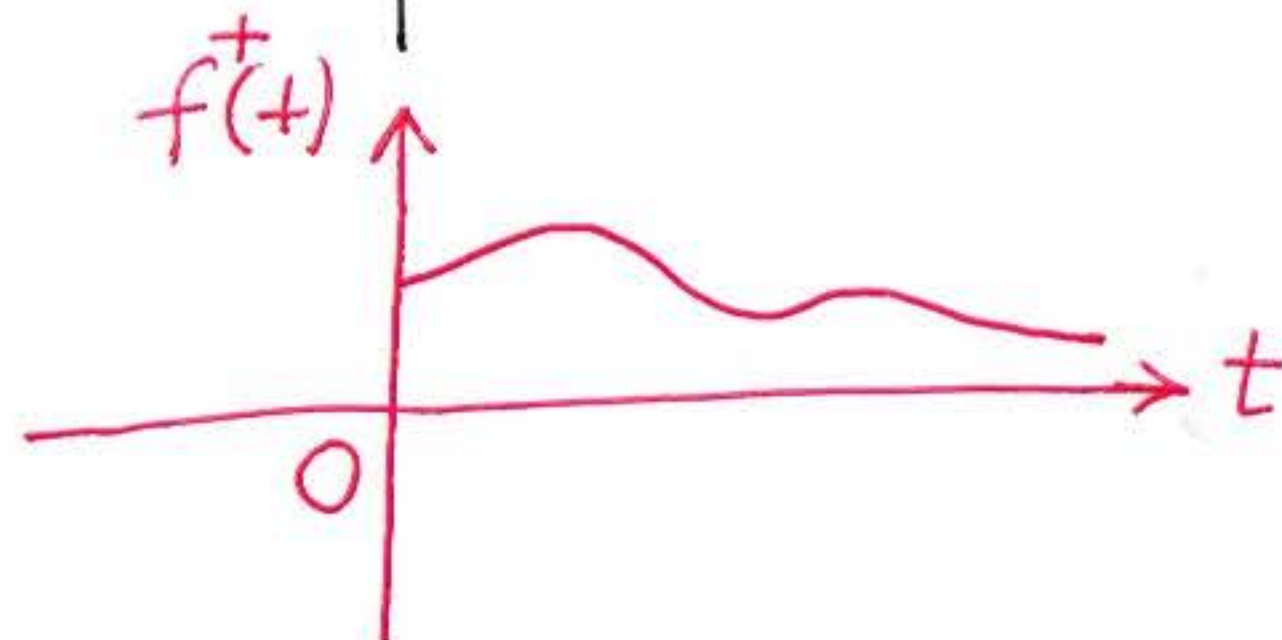
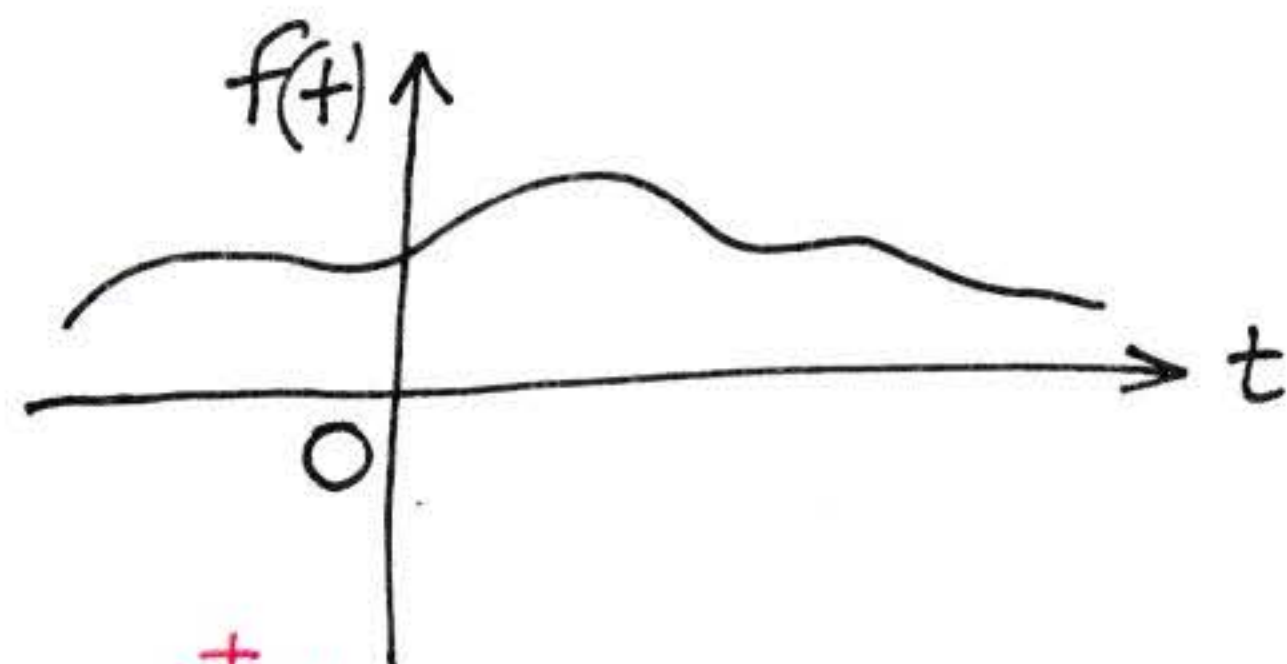
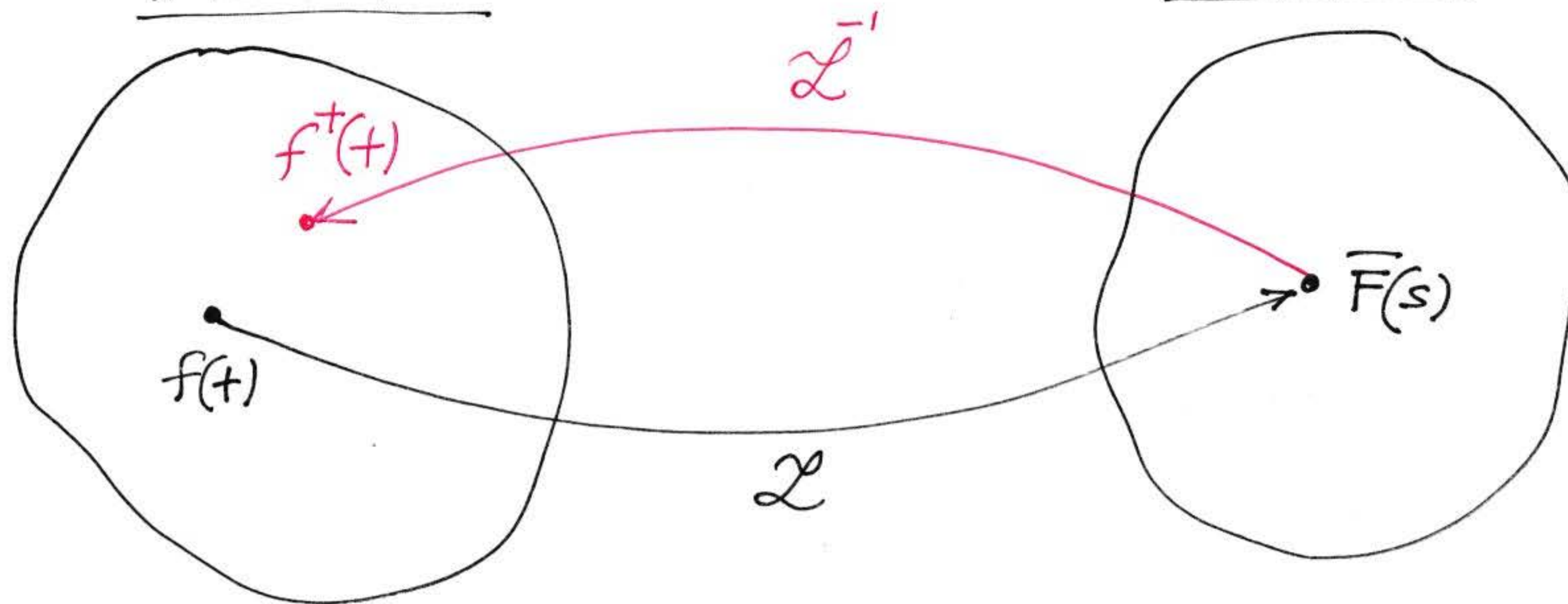


$$\mathcal{L}[f(t)] = \bar{F}(s), \quad \mathcal{L}[e^{-\alpha t} f(t)] = \bar{F}(s + \alpha), \quad \mathcal{L}^{-1}\left[\frac{A_k}{s - s_k}\right] = A_k e^{s_k t} \quad (t \geq 0)$$

$$\mathcal{L}^{-1}\left[\frac{A_k}{(s - s_k)^p}\right] = A_k \frac{t^{p-1}}{(p-1)!} e^{s_k t} \quad (t \geq 0), \quad \mathcal{L}^{-1}\left[\frac{B_1 s + B_2}{(s + \alpha)^2 + \omega^2}\right] = B_1 e^{-\alpha t} \cos \omega t + \frac{B_2 - \alpha B_1}{\omega} e^{-\alpha t} \sin \omega t \quad (t \geq 0)$$

Time-domain

\mathcal{L} -domain



$$\Rightarrow q(t) = \mathcal{L}^{-1}[\bar{q}(s)]$$

Convolution integral in terms of $\mathcal{L}^{-1}[\bar{R}(s)]$ and $\mathcal{L}^{-1}[\bar{G}(s)]$

i) General formulation

$[M]\{\ddot{q}\} + [K]\{q\} = \{Q(t)\}$ n -DOF oscillating system \Rightarrow Suppose that $\{q(t)\}$ and $\{Q(t)\}$ are \mathcal{L} -transformable $\Rightarrow \mathcal{L}[q_i(t)] = \int_0^\infty e^{-st} q_i(t) dt \equiv \bar{q}_i(s), i=1, \dots, n$

$\mathcal{L}[Q_i(t)] = \int_0^\infty e^{-st} Q_i(t) dt \equiv \bar{Q}_i(s), i=1, \dots, n$

Then \mathcal{L} -transform the equations of motion \Rightarrow

$$[M]\{s^2 \bar{q}(s) - s q(0+) - \dot{q}(0+)\} + [K]\{\bar{q}(s)\} = \{\bar{Q}(s)\} \Rightarrow$$

$$\Rightarrow [M]\{s^2 \bar{q}(s)\} + [K]\{\bar{q}(s)\} = \underbrace{\{\bar{Q}(s)\}}_{\text{Effect of external forcing}} + \underbrace{[M]s\{q(0+)\} + [M]\{\dot{q}(0+)\}}_{\text{Effect of initial conditions}} \underbrace{\}_{\text{Generalized transformed input}}_{\text{Transfer function } [G(s)]}$$

$$\underbrace{(s^2[M] + [K])\{\bar{q}(s)\}}_{\text{Generalized Impedance Matrix}} = \{\bar{Q}(s)\}$$

So, we write the equations of motion in the \mathcal{L} -domain as,

$$[Z(s)]\{\bar{q}(s)\} = \{\bar{R}(s)\} \Rightarrow \{\bar{q}(s)\} = [Z(s)]^{-1}\{\bar{R}(s)\}$$

So, we get the relation,

$$\{\bar{q}(s)\} = [Q(s)] \{\bar{R}(s)\} \quad (\mathcal{L}\text{-domain})$$

\uparrow \uparrow \uparrow
 output Transfer matrix input

Let us assume that $\{Q(s)\} = \{I\}$.

Hence, the transformed output is

$$\{\bar{q}(s)\} = \frac{[A(s)]}{|Z(s)|} [M] \{s\dot{q}_0 + \dot{q}_0\}$$

\uparrow \uparrow \uparrow
 Adjoint matrix of $[Z(s)]$ Determinant of $[Z(s)]$ $\{q(0+)\}$

But note that the denominator $|Z(s)|$ can be expressed in terms of the natural frequencies of the unforced problem,

$$|Z(s)| = \underbrace{(s^2 + \omega_1^2)}_{(s - j\omega_1)(s + j\omega_1)} \dots \underbrace{(s^2 + \omega_n^2)}_{(s - j\omega_n)(s + j\omega_n)},$$

$j = (-1)^{1/2}$, and we assume that the unforced system has distinct eigenfrequencies $\Rightarrow |Z(s)| = \prod_{r=1}^n (s^2 + \omega_r^2) \Rightarrow \{\bar{q}(s)\} = \frac{[A(s)]}{\prod_{r=1}^n (s^2 + \omega_r^2)} [M] \{s\dot{q}_0 + \dot{q}_0\} \Rightarrow$

$\Rightarrow \{q(t)\} = \mathcal{L}^{-1} \{\bar{q}(s)\} \Rightarrow$ We use the method of partial fraction expansion \Rightarrow

$$\text{Then, } \{q(t)\} = \sum_{k=1}^n \left\{ (s - j\omega_k) \{ \bar{q}(s) \} e^{st} \Big|_{s=j\omega_k} + (s + j\omega_k) \{ \bar{q}(s) \} e^{st} \Big|_{s=-j\omega_k} \right\} =$$

$$= \sum_{k=1}^n \left\{ \frac{[A(s)][M] \{s\dot{q}_0 + \ddot{q}_0\} e^{st}}{(s + j\omega_k) \prod_{\substack{r=1 \\ r \neq k}}^n (s^2 + j\omega_r^2)} \Big|_{s=j\omega_k} + \frac{[A(s)][M] \{s\dot{q}_0 + \ddot{q}_0\} e^{st}}{(s - j\omega_k) \prod_{\substack{r=1 \\ r \neq k}}^n (s^2 + j\omega_r^2)} \Big|_{s=-j\omega_k} \right\}$$

$$\text{But } [A(s=j\omega_k)] = [A(s=-j\omega_k)] \Rightarrow$$

Note that the response is expressed as superposition of the n modes!

$$\Rightarrow \{q(t)\} = \sum_{k=1}^n \frac{[A(j\omega_k)][M]}{\prod_{\substack{r=1 \\ r \neq k}}^n (\omega_r^2 - \omega_k^2)} \left\{ \{q_0\} \cos \omega_k t + \{\dot{q}_0\} \frac{\sin \omega_k t}{\omega_k} \right\}$$

Response due to initial conditions, under the assumption of distinct natural frequencies and no damping

$$\bar{q}(s) = \frac{[A(s)]}{(s^2 + \omega_1^2)(s^2 + \omega_2^2) \dots (s^2 + \omega_n^2)} =$$

$$= \frac{[A(s)]}{(s - j\omega_1)(s + j\omega_1)(s - j\omega_2)(s + j\omega_2) \dots (s - j\omega_n)(s + j\omega_n)}$$

$$\{q(t)\} = \sum_{k=1}^n \left[\left\{ \frac{(s - j\omega_k) [A(s)] e^{st}}{(s - j\omega_1) \dots (s - j\omega_k) (s + j\omega_k) \dots (s + j\omega_n)} \right\}_{s=j\omega_k} + \right. \\ \left. + \left\{ \frac{(s + j\omega_k) [A(s)] e^{st}}{\dots (s - j\omega_k) (s + j\omega_k) \dots} \right\}_{s=-j\omega_k} \right]$$

ii) Response to harmonic excitation

Suppose now that $\{Q(t)\} = [Q_1(t) \dots Q_n(t)]^T = [Q_{01} e^{j\Omega_1 t} \dots Q_{0n} e^{j\Omega_n t}]^T \Rightarrow$
 Complex amplitudes

\Rightarrow \mathcal{L} -transform the forcing vector $\{\bar{Q}(s)\} = \left[\frac{Q_{01}}{s-j\Omega_1} \dots \frac{Q_{0n}}{s-j\Omega_n} \right]^T \equiv \left\{ \frac{Q_0}{s-j\Omega} \right\}$

Assuming zero initial conditions $\Rightarrow \{q_0\} = \{\dot{q}_0\} = \{0\} \Rightarrow$
 $\leftarrow [G(s)] = [Z(s)]^{-1}$

$\Rightarrow \{\bar{q}(s)\} = \frac{[A(s)]}{|Z(s)|} \left[\frac{Q_{01}}{s-j\Omega_1} \dots \frac{Q_{0n}}{s-j\Omega_n} \right]^T \Rightarrow$

Now, let's express the adjoint matrix $[A(s)]$ in terms of its columns \Rightarrow

$\Rightarrow [A(s)] = [\{A_1(s)\} \dots \{A_n(s)\}]$

$\Rightarrow \{\bar{q}(s)\} = \sum_{k=1}^n \frac{\{A_k(s)\}}{|Z(s)|} \frac{Q_{0k}}{s-j\Omega_k}$

\leftarrow Has 2 poles the imaginary eigenvalues $\pm j\omega_k$
 \rightarrow Additional poles introduced by the external harmonic forces!

The roots of $|Z(s)|=0$ are the poles $s=\pm j\omega_r, r=1, \dots, n$, which contribute to the transient response of the system which is zero for zero initial conditions; however, the contributions of the poles $s=j\Omega_k$ resulting from the external harmonic excitations contribute to the steady state response (which does not depend on the initial conditions of the problem) \Rightarrow Hence, we compute the steady state response as,

$$\{q(t)\} = \sum_{k=1}^n \frac{\{A_k(j\Omega_k)\}}{|Z(j\Omega_k)|} Q_{0k} e^{j\Omega_k t} \quad (\text{steady-state response})$$

provided that $\Omega_k \neq \omega_m \quad \forall k, m \in [1, \dots, n]$ (lack of resonance). We note that when $\Omega_k \rightarrow \omega_m$ for some $k, m \in [1, \dots, n] \Rightarrow |Z(j\Omega_k)| \rightarrow 0$ and $\|\{q(t)\}\| \rightarrow \infty \Rightarrow$ In such a case the previous inversion cannot be performed and an alternative inversion must be followed that takes into account the resonance condition, i.e., $\Omega_k = \omega_m$ (linear resonance).

In the case of resonance, we get \rightarrow When $\omega_m = \Omega_k$

$$\{g(t)\} = \frac{d}{ds} \left[(s - j\omega_m)^2 \frac{\{A_k(s)\}}{|Z(s)|} \frac{Q_{0k}}{s - j\Omega_k} e^{st} \right] \bigg|_{s=j\omega_m} +$$

$$+ \sum_{\substack{p=1 \\ p \neq k}}^n \frac{\{A_p(j\Omega_p)\}}{|Z(j\Omega_p)|} Q_{0p} e^{j\Omega_p t} =$$

Note that this term explodes as $t \rightarrow \infty$

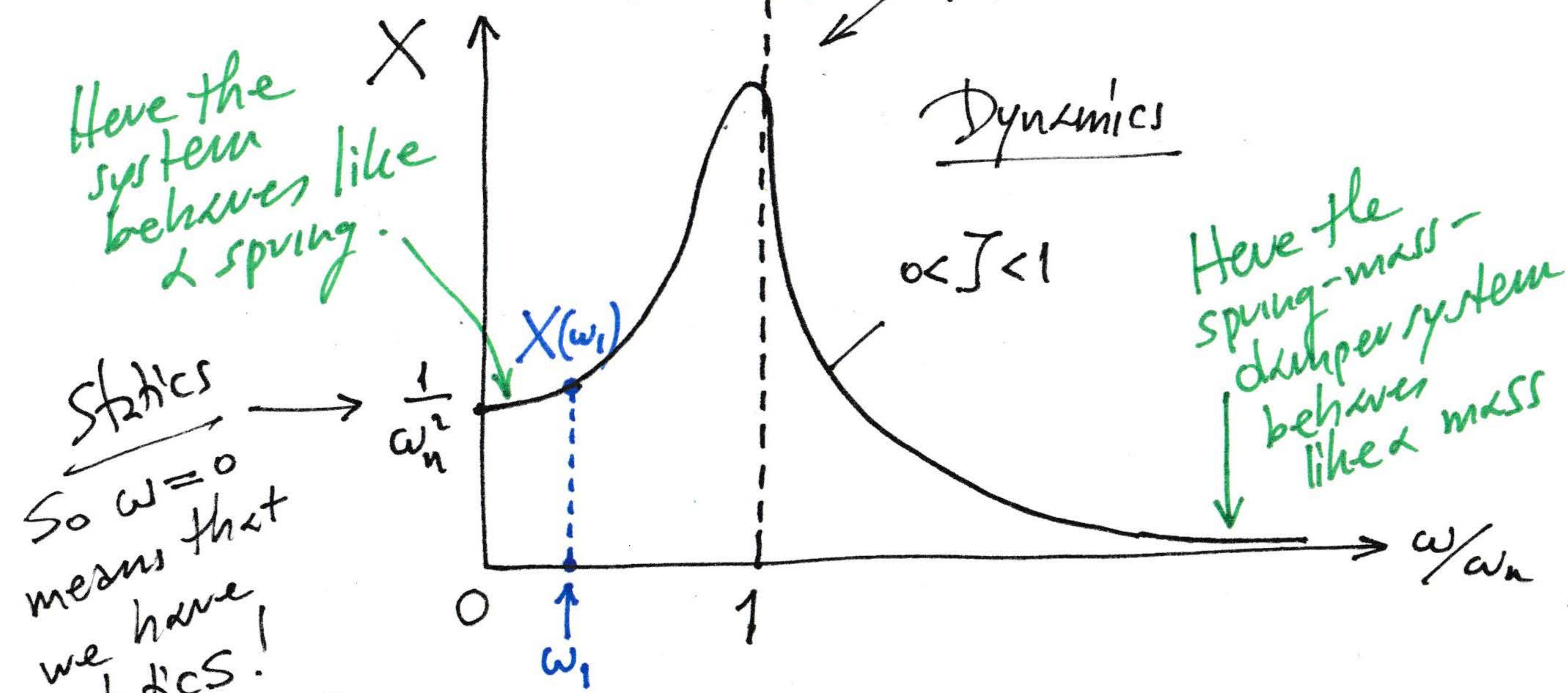
$$= \frac{Q_{0k} e^{j\Omega_k t}}{2j\Omega_k \sum_{\substack{r=1 \\ r \neq m}}^n (\omega_r^2 - \Omega_k^2)} \left\{ t A_k(j\Omega_k) + A'_k(j\Omega_k) - \left(\frac{1}{2j\Omega_k} + \sum_{\substack{r=1 \\ r \neq m}}^n \frac{2j\Omega_k}{\omega_r^2 - \Omega_k^2} \right) A_k(j\Omega_k) \right\} +$$

$$+ \sum_{\substack{p=1 \\ p \neq k}}^n \frac{\{A_p(j\Omega_p)\}}{|Z(j\Omega_p)|} Q_{0p} e^{j\Omega_p t}$$

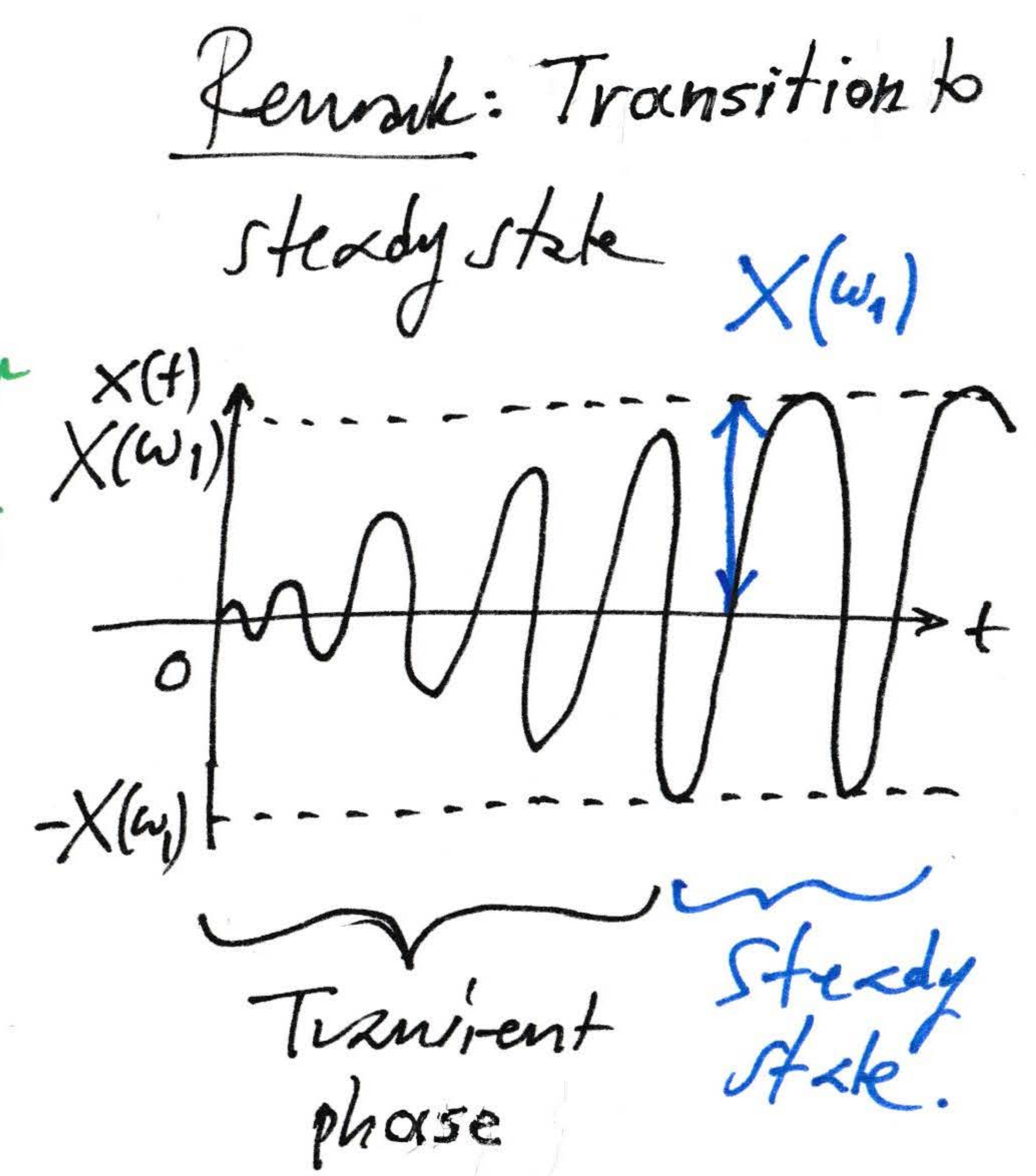
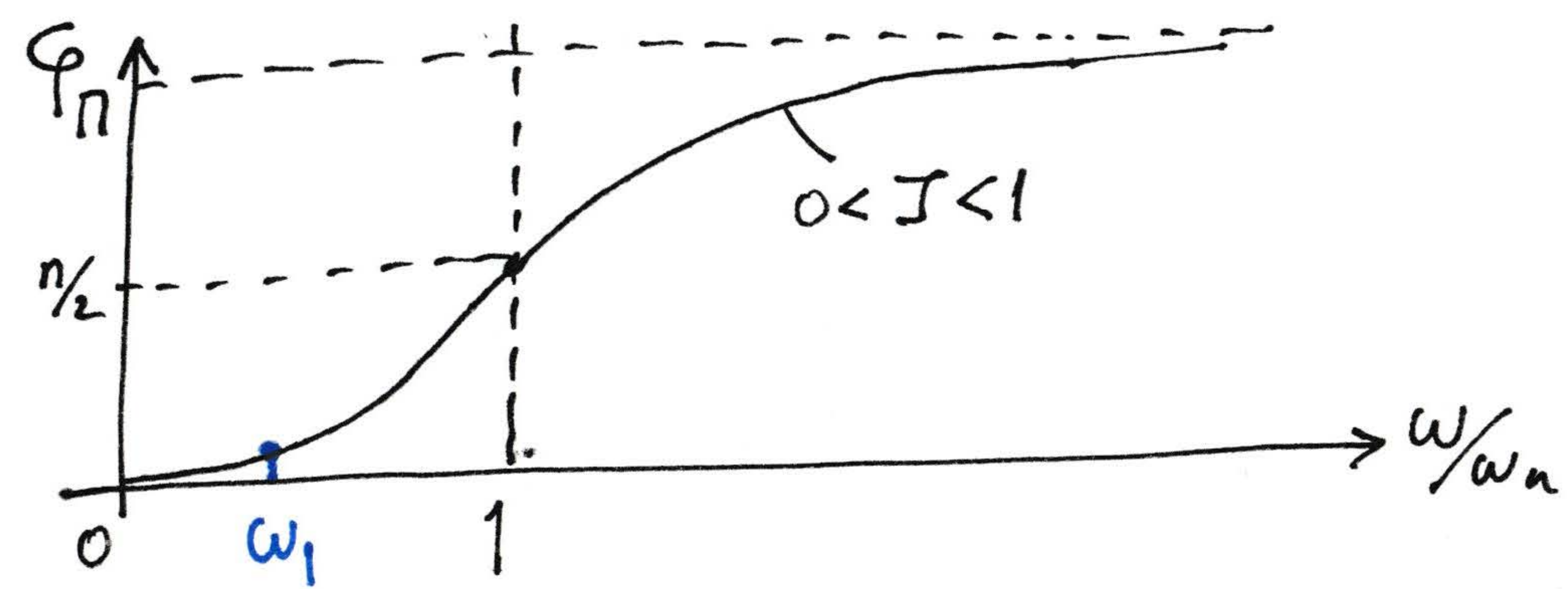
Frequency response - resonance

$\ddot{X} + 2\zeta\omega_n \dot{X} + \omega_n^2 X = P \cos \omega t \rightarrow$ SS solution (not interested in ICs)

$X_{ss}(t) = X \cos(\omega t - \phi)$ Always positive (modulus)
Phase lag with respect to the excitation
 Note that, e.g., if $\phi = 0 \Rightarrow X_{ss}(t) = X \cos \omega t$
 if $\phi = \pi \Rightarrow X_{ss}(t) = -X \cos \omega t$

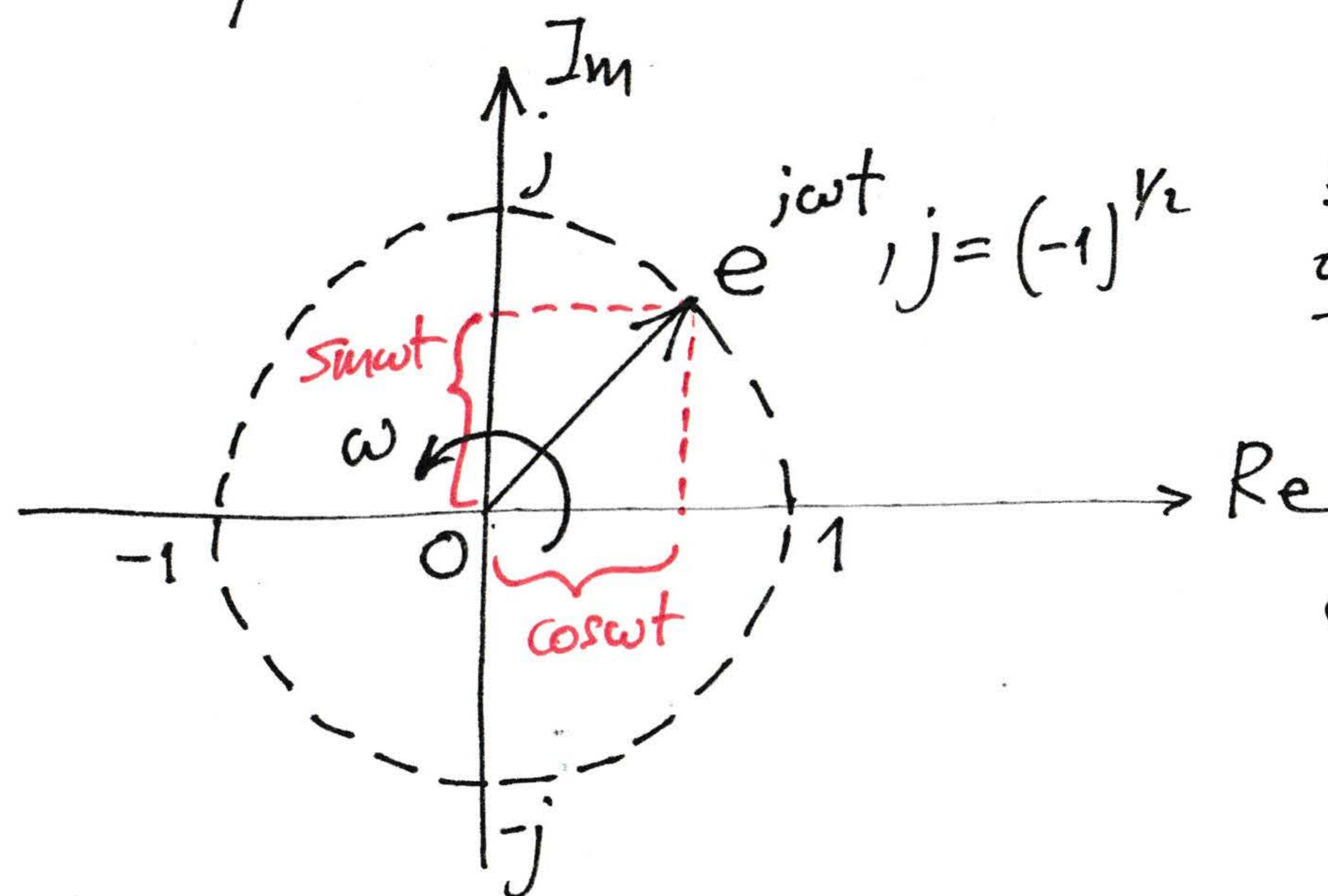


Statics
 So $\omega = 0$ means that we have statics!
 Whereas $\omega > 0$ means dynamics.



So, the resonance plot depicts only the steady state response (and ignores the transient phase)

We now introduce a complex vector representation to simplify the analysis.



$$e^{j\omega t} = \cos \omega t + j \sin \omega t$$

$$\cos \omega t = \text{Re}[e^{j\omega t}]$$

$$\sin \omega t = \text{Im}[e^{j\omega t}]$$

So, a harmonic motion is represented as a rotating vector (of constant magnitude) in the complex domain.

So, reconsider,

Real domain

$$\ddot{x} + 2\zeta\omega_n \dot{x} + \omega_n^2 x = \begin{cases} P \sin \omega t \\ P \cos \omega t \end{cases}$$

↓ Replace by an equivalent problem

Complex domain

$$\ddot{x} + 2\zeta\omega_n \dot{x} + \omega_n^2 x = P e^{j\omega t}$$

Solve this problem and then "go back" to the real domain.

Suppose that the solution in the complex domain is $x_{ss}^c(t)$.

Recognizing that $P \sin \omega t = \text{Im}[P e^{j\omega t}]$

$$x_{ss}^R(t) = \text{Im}[x_{ss}^c(t)] \quad \leftarrow \boxed{x_{ss}^c(t)}$$

If, $P \cos \omega t = \text{Re}[P e^{j\omega t}]$

$$x_{ss}^R(t) = \text{Re}[x_{ss}^c(t)] \quad \leftarrow \boxed{x_{ss}^c(t)}$$

Digression: Suppose that we have a complex function, $\frac{a+jb}{c+jd} \Rightarrow$ How do we compute its modulus and phase?

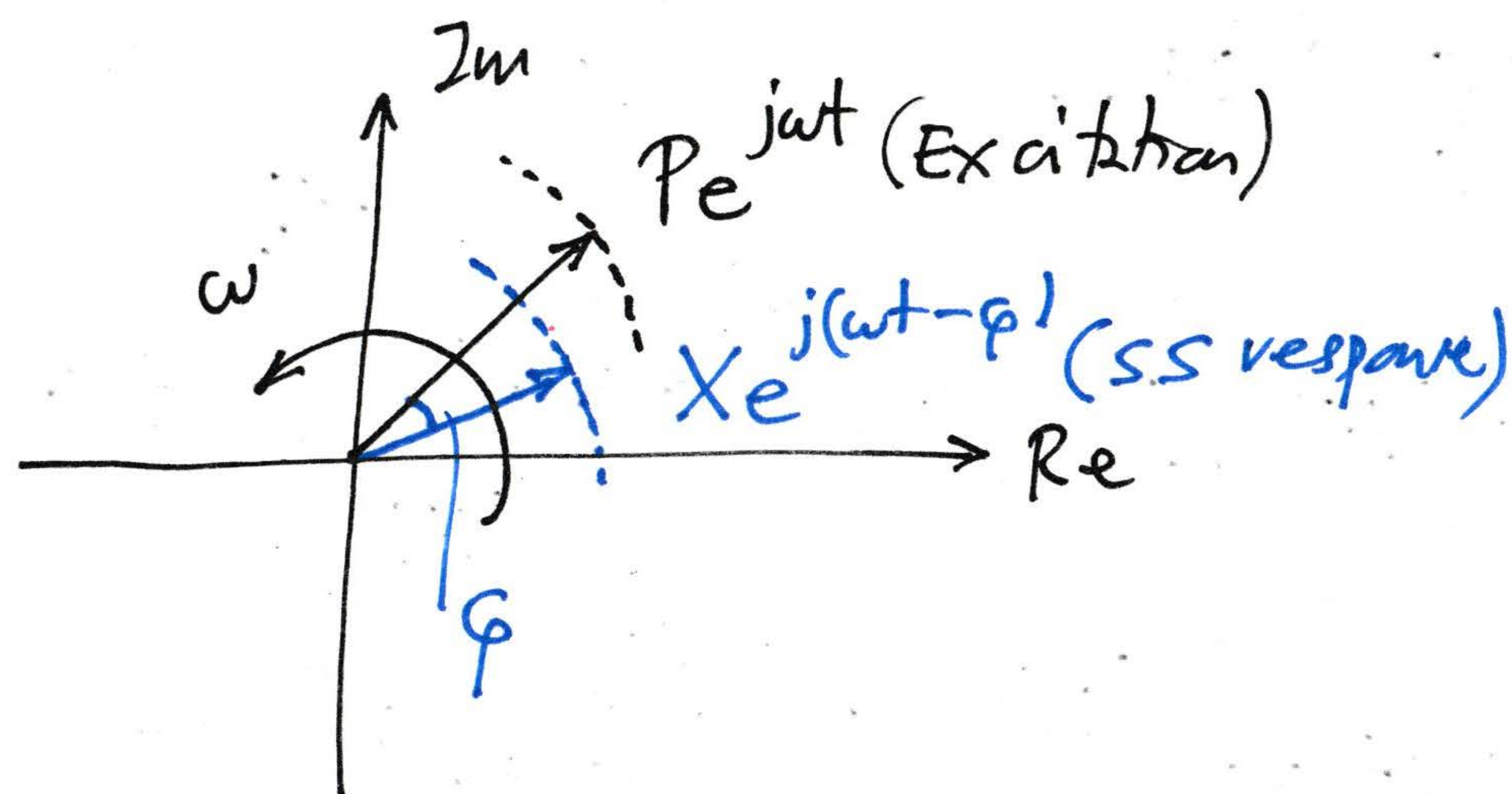
$$\frac{a+jb}{c+jd} = \frac{(a+jb)(c-jd)}{(c+jd)(c-jd)} = \frac{(ac+bd) + j(bc-ad)}{c^2+d^2} =$$

$$= \underbrace{\left(\frac{ac+bd}{c^2+d^2} \right)}_{\text{Real part (Re)}} + j \underbrace{\left(\frac{bc-ad}{c^2+d^2} \right)}_{\text{Imaginary part (Im)}} \Rightarrow$$

$$\Rightarrow \text{Modulus} = \sqrt{\text{Re}^2 + \text{Im}^2} = \dots$$

$$\text{Phase is } \tan^{-1} \left(\frac{\text{Im}}{\text{Re}} \right) = \dots$$

To find the SS solution in the complex domain, look for a solution of the form, $X_{ss}(t) = X e^{j(\omega t - \phi)} = X e^{j\omega t} e^{-j\phi} =$



Real magnitude > 0 $= X^* e^{j\omega t} \quad (*)$
 where $X^* = X e^{-j\phi}$,
 X^* is complex amplitude

Substituting $(*)$ into the complex problem,

$$-\omega^2 X^* e^{j\omega t} + 2j\omega_n j\omega X^* e^{j\omega t} + \omega_n^2 X^* e^{j\omega t} = P e^{j\omega t} \Rightarrow$$

\Rightarrow Solve directly for $X^* = \frac{P}{(\omega_n^2 - \omega^2) + j2j\omega_n\omega} \Rightarrow$

$$\Rightarrow \boxed{\frac{X^*}{P} = \frac{1}{\omega_n^2 - \omega^2 + j2j\omega_n\omega}}$$

Steady state response $(**)$

Applying this methodology to $(*) \Rightarrow$

$$\frac{X}{P} = \frac{|X^*|}{P} = \left(\frac{1}{\omega_n^2} \right) \frac{1}{\left\{ \left[1 - \left(\frac{\omega}{\omega_n} \right)^2 \right]^2 + \left(2 \zeta \frac{\omega}{\omega_n} \right)^2 \right\}^{1/2}}$$

$$\phi = \tan^{-1} \left\{ \frac{2 \zeta \omega / \omega_n}{1 - \left(\frac{\omega}{\omega_n} \right)^2} \right\}$$

Recap: The solution in the complex domain is

Solution in the complex plane (also denoted as X_{ss})

$$\longrightarrow X_{ss}(t) = X^* e^{j\omega t} = X e^{j(\omega t - \phi)}$$

↓ To the real domain

Solution in the real plane (also denoted as $X_{ss}^c(t)$)

If excitation is $P \cos \omega t \Rightarrow X_{ss}(t) = \text{Re} [X e^{j(\omega t - \phi)}] =$

If excitation is $P \sin \omega t \Rightarrow X_{ss}(t) = \text{Im} [X e^{j(\omega t - \phi)}] =$

$$= X \cos(\omega t - \phi)$$

$$= X \sin(\omega t - \phi)$$

At steady state we have the following dynamic equilibrium:

$$\ddot{X}_{ss} + 2j\omega_n \dot{X}_{ss} + \omega_n^2 X_{ss} = P e^{j\omega t}$$

Inertia force Damping force Stiffness force Excitation force

Given that $X_{ss}(t) = X e^{j(\omega t - \phi)} = \underbrace{X}_{X^*} e^{j\omega t}$

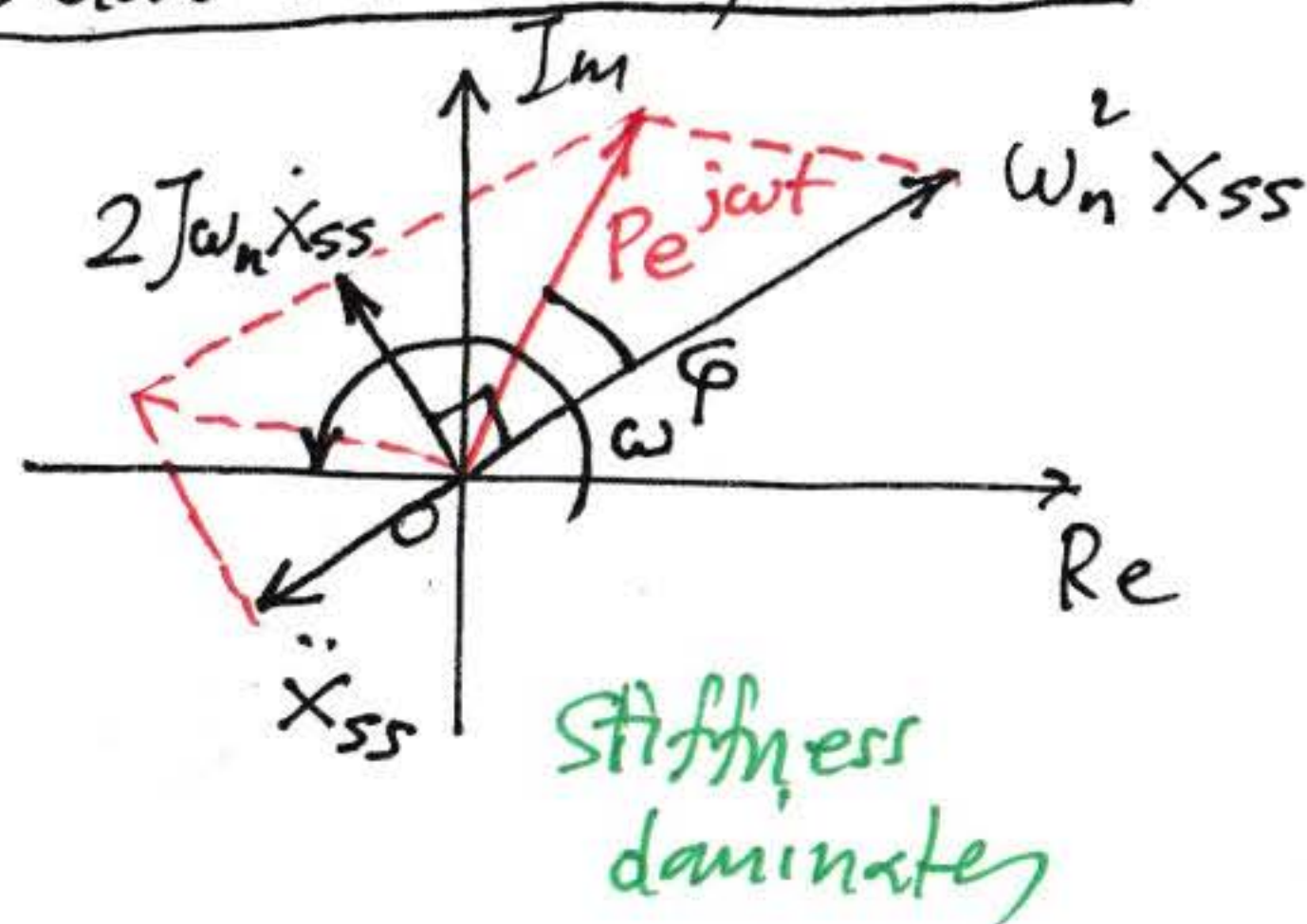
$$\omega_n^2 X_{ss}(t) = \omega_n^2 X^* e^{j\omega t}$$

$$\ddot{X}_{ss} = -\omega^2 X^* e^{j\omega t}$$

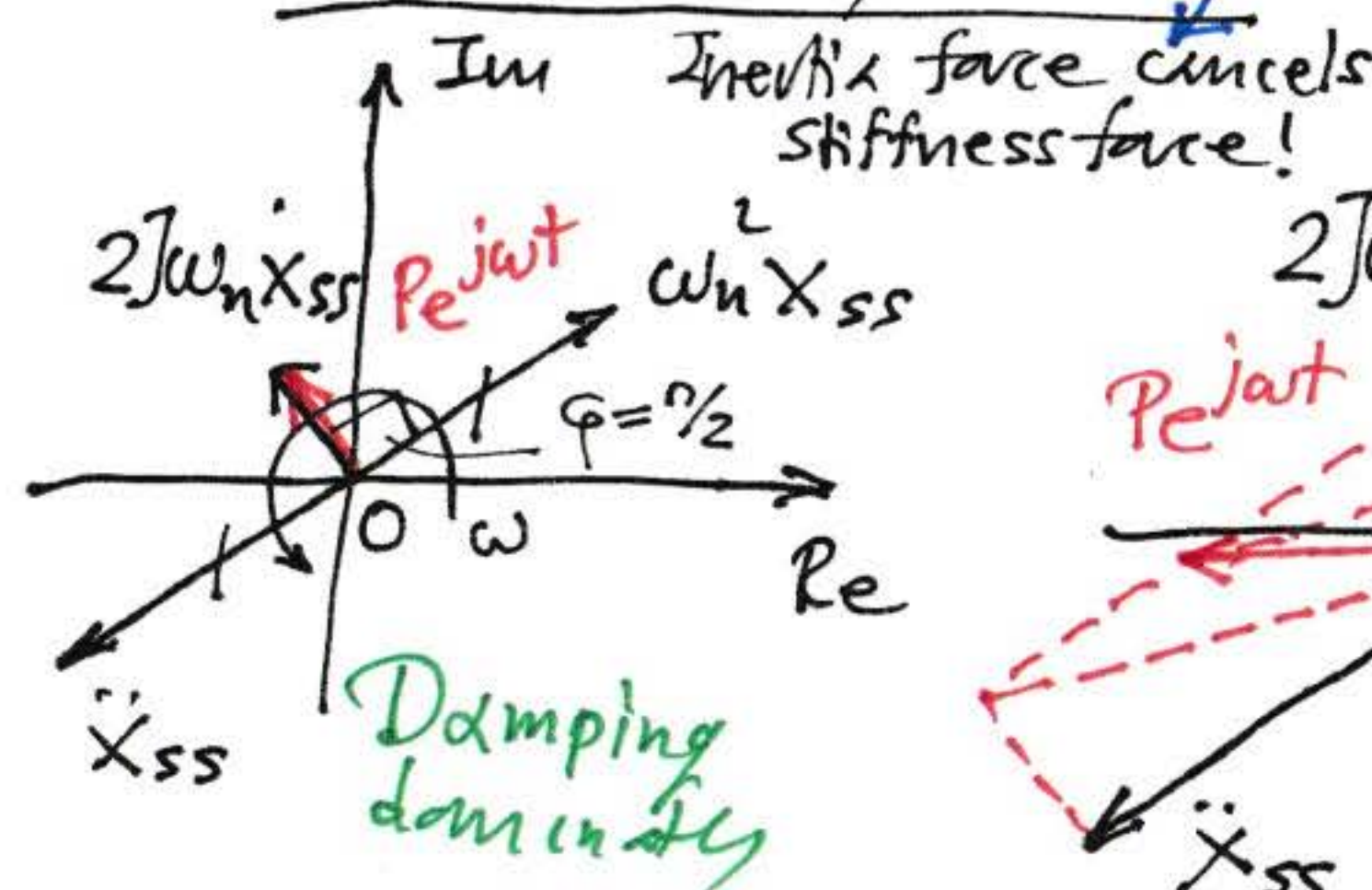
$$2j\omega_n \dot{X}_{ss} = 2j\omega_n \omega X^* e^{j\omega t}$$

These are rotating vectors in the complex plane!

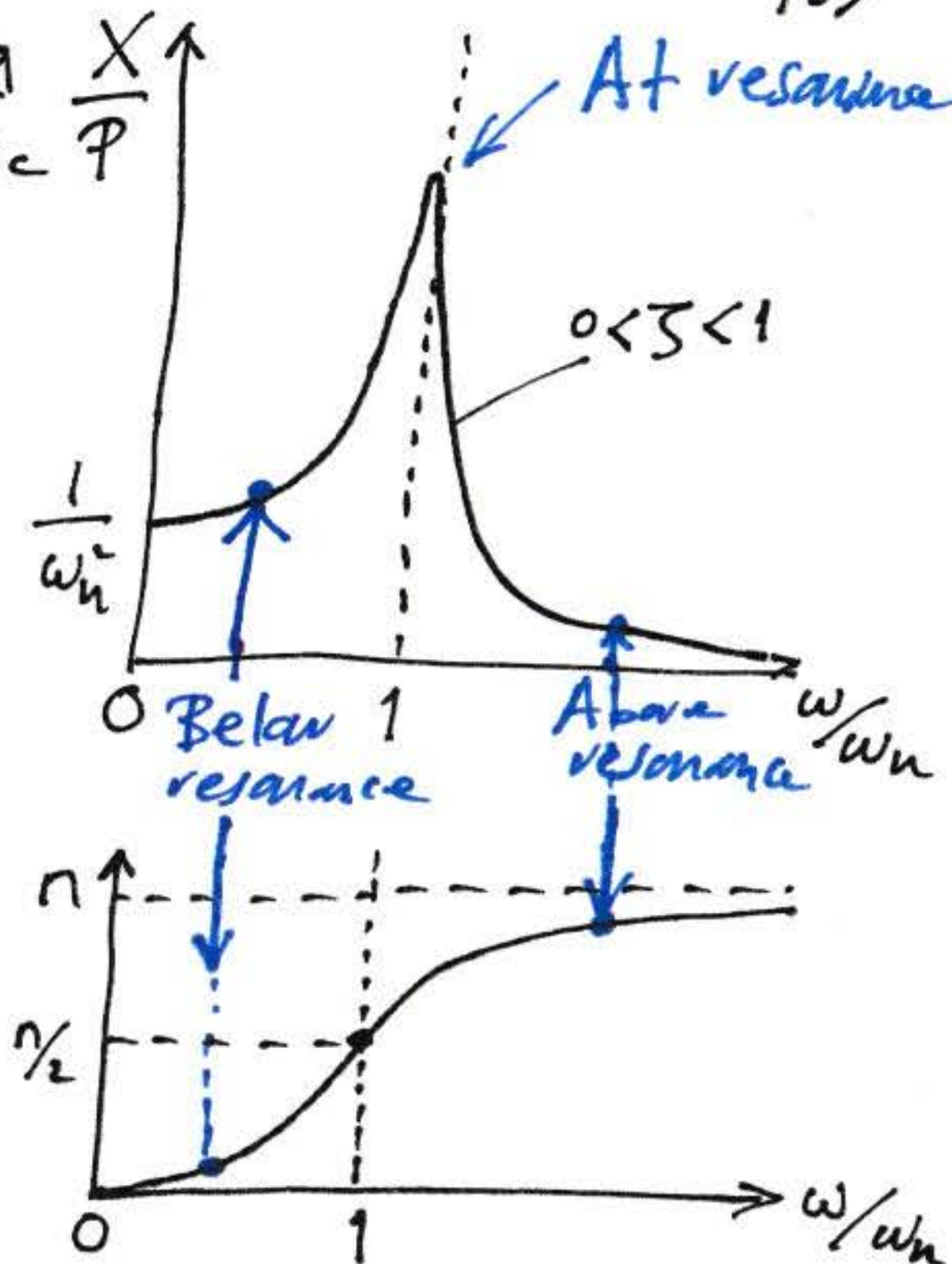
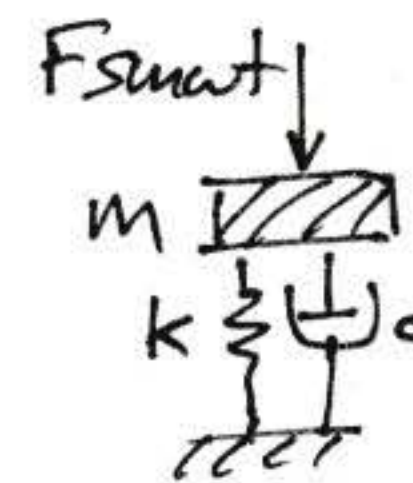
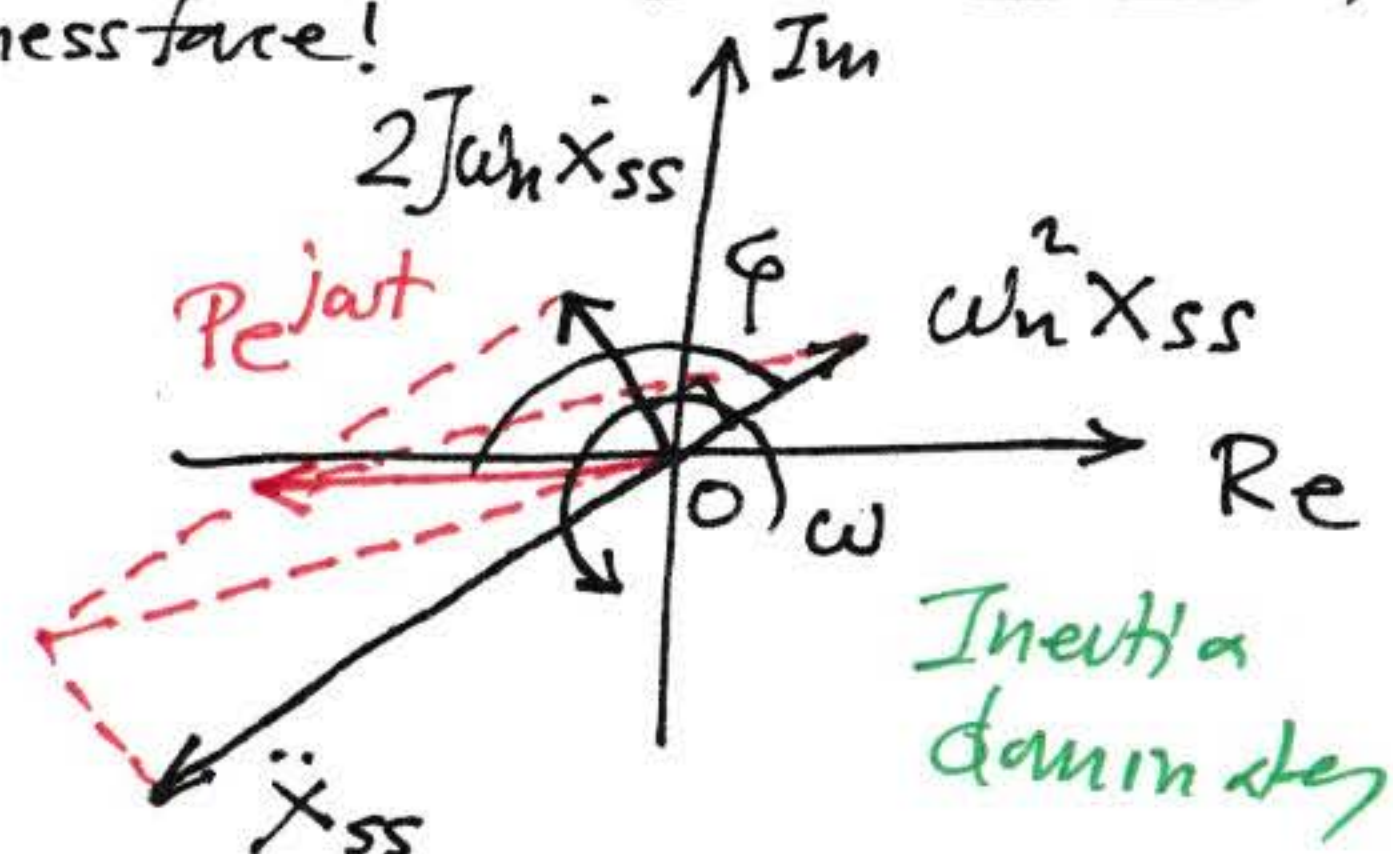
Below resonance, $\omega < \omega_n$



At resonance, $\omega = \omega_n$



Above resonance, $\omega > \omega_n$



Remarks

- 1) In the undamped system at resonance the external harmonic excitation cannot be balanced by the inertia or stiffness forces which cancel each other. \Rightarrow We get infinite steady state response at resonance!

This can be concluded by solving the initial value (transient) problem as follows:

Consider, $\ddot{x} + \omega_n^2 x = P \cos \omega_n t$ \Rightarrow The solution is the particular solution,
 $x(0) = \dot{x}(0) = 0$ $x(t) = \frac{Pt}{\omega_n} \sin \omega_n t$

So, by solving the transient problem at resonance we recover the uncontrollable growth of the response to infinity!

