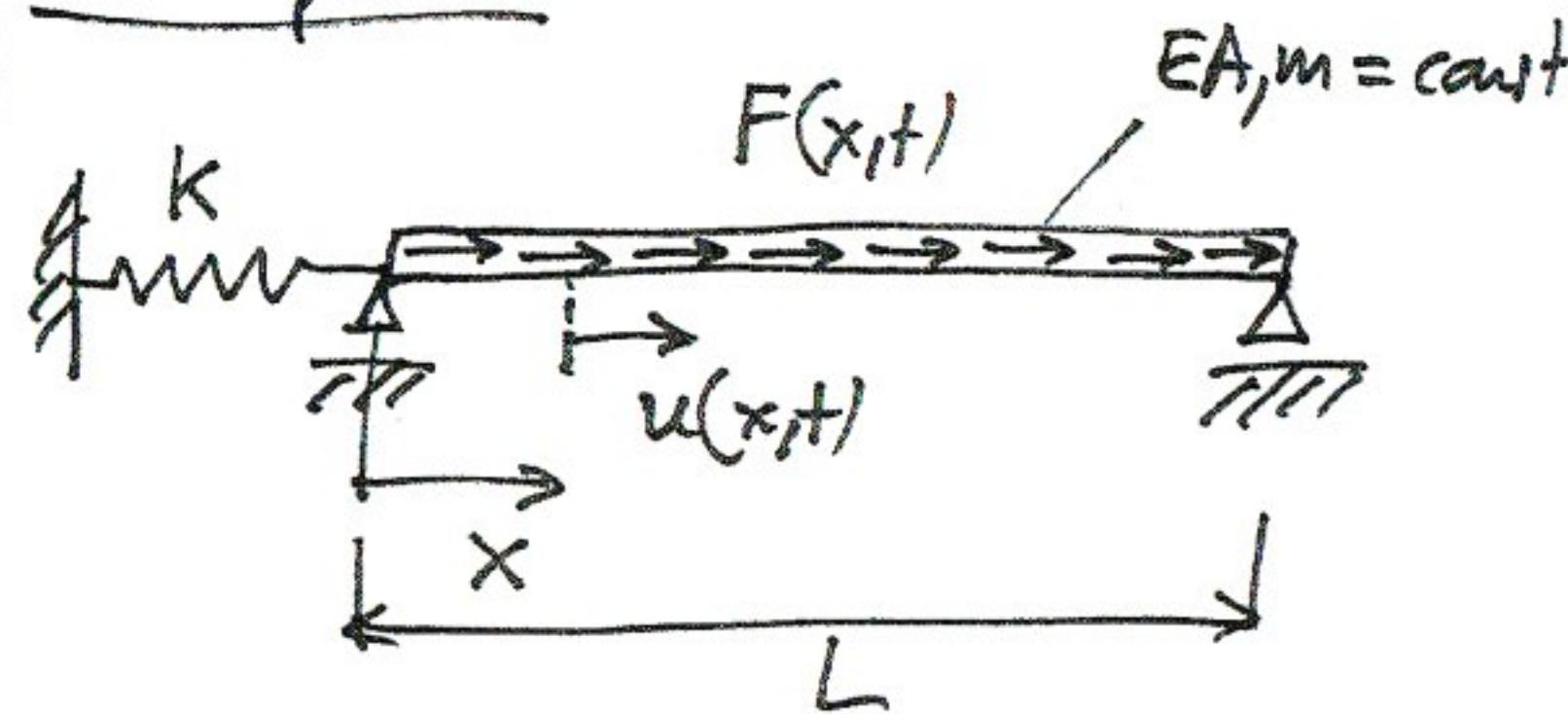


Example 3

$$EA \frac{\partial^2 u}{\partial x^2} + F(x,t) = m \frac{\partial^2 u}{\partial t^2}, \quad 0 \leq x \leq L \quad (1)$$

$t \geq 0$

$$\left. \begin{aligned} EA \frac{\partial u(0,t)}{\partial x} - ku(0,t) &= 0 \\ \frac{\partial u(L,t)}{\partial x} &= 0 \end{aligned} \right\} \text{BCs} \quad (1a)$$

$$u(x,0) = g(x), \quad \frac{\partial u}{\partial t}(x,0) = h(x) \quad (1b)$$

first we compute the modes of this system  $\Rightarrow \varphi''(x) + \left(\frac{\omega}{c}\right)^2 \varphi(x) = 0 \quad (2)$

$$\left. \begin{aligned} EA \varphi'(0) - k \varphi(0) &= 0 \\ \varphi'(L) &= 0 \end{aligned} \right\} \quad (2a)$$

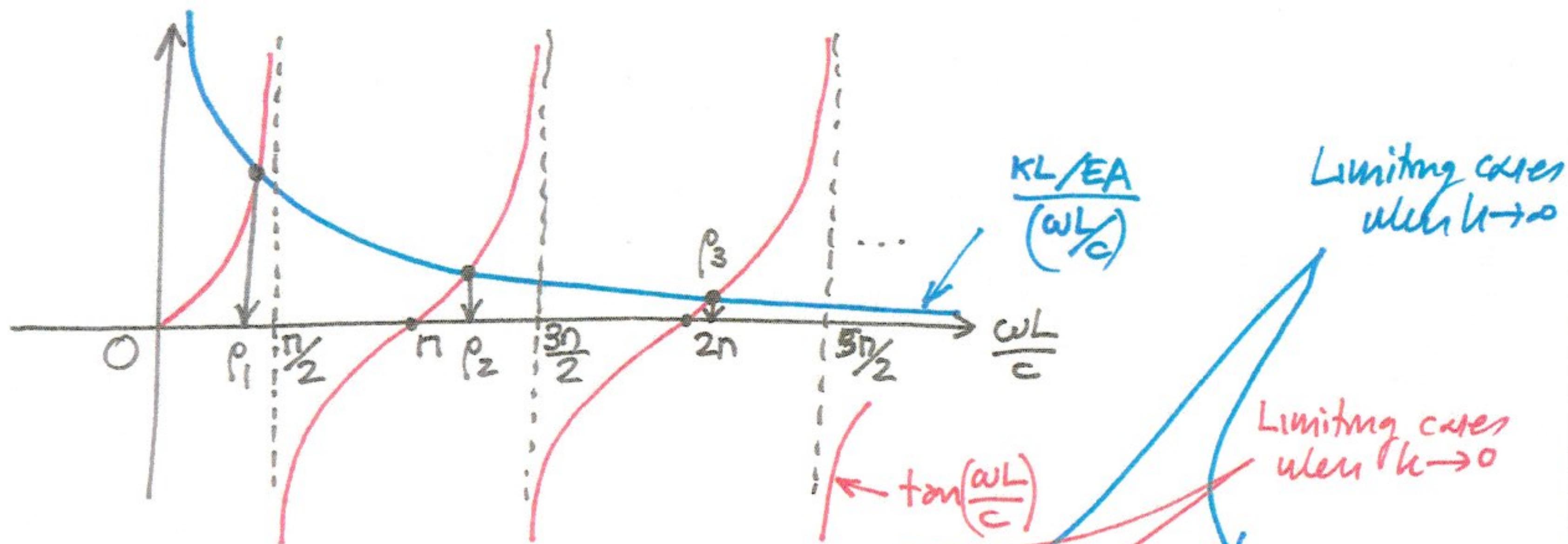
Solving for  $\varphi(x) \Rightarrow \varphi(x) = C_1 \cos \frac{\omega x}{c} + C_2 \sin \frac{\omega x}{c} \Rightarrow$  Applying (2a)  $\Rightarrow$

$$\left. \begin{aligned} EA C_2 \frac{\omega}{c} - k C_1 &= 0 \\ -C_1 \frac{\omega}{c} \sin \frac{\omega L}{c} + C_2 \frac{\omega}{c} \cos \frac{\omega L}{c} &= 0 \end{aligned} \right\} \Rightarrow \begin{bmatrix} -k & EA \frac{\omega}{c} \\ -\sin \frac{\omega L}{c} & \cos \frac{\omega L}{c} \end{bmatrix} \begin{Bmatrix} C_1 \\ C_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \Rightarrow$$

$\Rightarrow$  for nontrivial solutions we require that  $\det \left[ \begin{array}{cc} -k & EA \frac{\omega}{c} \\ -\sin \frac{\omega L}{c} & \cos \frac{\omega L}{c} \end{array} \right] = 0 \Rightarrow$

$$\Rightarrow -k \cos \frac{\omega L}{c} + EA \frac{\omega}{c} \sin \frac{\omega L}{c} = 0 \Rightarrow \tan \left( \frac{\omega L}{c} \right) = \frac{k/EA}{\omega/c} = \frac{k/EA}{(\omega/c)L} \Rightarrow \tan \left( \frac{\omega L}{c} \right) = \frac{KL}{(\omega/c)L}$$

Frequency equation



So we get a countable infinity of roots  $0 < \rho_1 < \frac{\pi}{2}, \pi < \rho_2 < \frac{3\pi}{2}, \dots \Rightarrow$

$\Rightarrow$  Then we obtain the corresponding natural frequencies  $\leftrightarrow \frac{\omega_i L}{c} = \rho_i \Rightarrow$

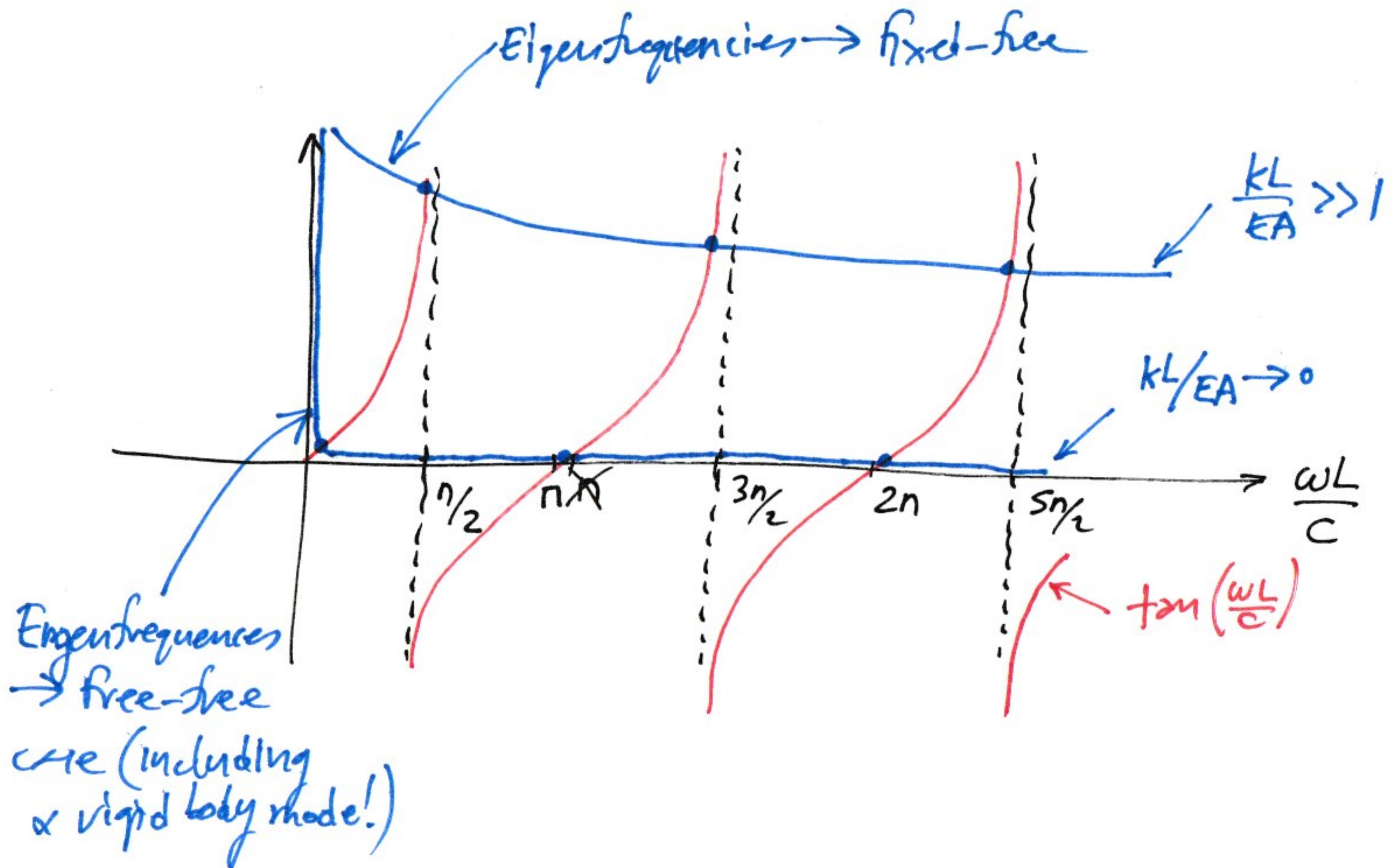
$\Rightarrow \omega_i = \frac{c \rho_i}{L}, i=1,2,\dots \Rightarrow$  Then we can compute the corresponding eigenfunctions

by noting that  $c_2 = c_1 \tan \frac{\omega_i L}{c} \Rightarrow \varphi_i(x) = c_i \left( \cos \frac{\omega_i x}{c} + \tan \frac{\omega_i L}{c} \sin \frac{\omega_i x}{c} \right) \Rightarrow$

$$\Rightarrow \varphi_i(x) = c_i \frac{\cos(\rho_i \frac{L-x}{L})}{\cos \rho_i}$$

Eigenfunctions for  $i=1,2,\dots$ . Of course, we may

orthonormalize the eigenfunctions  $\varphi_i(x)$  using the formulas for 'non-simple' BCs derived in earlier lecture.





To perform orthonormalization we need that for each eigenfunction  $\varphi_i(x)$  it holds that,

$$\int_0^L m \varphi_i^2(x) dx = 1, i=1,2,\dots \Rightarrow$$



In simple form since the mass is uniform

$$\Rightarrow \int_0^L m c_i^2 \frac{\cos^2(\rho_i \frac{L-x}{L})}{\cos^2 \rho_i} dx = 1 \Rightarrow$$



$$\Rightarrow c_i = 2 \cos \rho_i \sqrt{\frac{\rho_i}{Lm(2\rho_i + \sin 2\rho_i)}} \\ i=1,2,\dots$$

Then, the orthonormalized eigenfunctions are,

$$\boxed{\varphi_i(x) = 2 \sqrt{\frac{\rho_i}{Lm(2\rho_i + \sin 2\rho_i)}} \cos \rho_i \frac{L-x}{L}, \quad \omega_i = \rho_i \frac{\pi}{L}, \quad i=1,2,\dots}$$

Note that the stiffness-orthogonality condition won't be simple due to the local stiffness at  $x=0$ .

Remark: What happens when  $k \rightarrow 0$ ? Suppose that  $\frac{UL}{EA} \ll 1 \Rightarrow$

$$\Rightarrow \tan\left(\frac{\omega L}{c}\right) = \frac{KL/EA}{\left(\frac{\omega L}{c}\right)} \Rightarrow \tan \frac{\omega L}{c} \ll 1 \Rightarrow \tan \frac{\omega L}{c} \sim \frac{\omega L}{c} \Rightarrow$$

For the first branch of  $\tan$

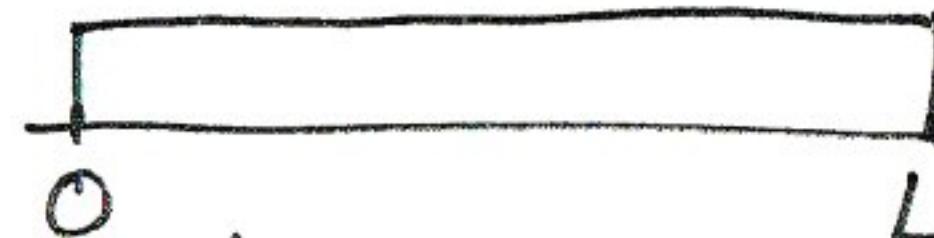
$$\Rightarrow \frac{\omega_1 L}{c} \sim \frac{UL/EA}{\frac{\omega_1 L}{c}} \Rightarrow \left(\frac{\omega_1 L}{c}\right)^2 \sim \frac{KL}{EA} \Rightarrow \omega_1^2 \sim \frac{c^2}{L^2} \frac{KL}{EA} = \frac{EA}{ML} \frac{K}{EA} = \frac{K}{mL} \Rightarrow$$

$\Rightarrow \omega_1^2 \sim \frac{K}{mL}$

Then we get the approximation that the rod moves as a rigid body

around  So as  $K \rightarrow 0 \Rightarrow \omega_1^2 \rightarrow 0 \Rightarrow \varphi_1(x) = \text{const}$

$$\uparrow M = mL$$



Hence the first mode becomes the rigid-body mode.

But what happens to higher modes? As  $k \rightarrow 0 \Rightarrow \tan \frac{\omega L}{c} \rightarrow 0 \Rightarrow$

$\leftarrow$  for other branches of  $\tan$

$$\Rightarrow \frac{\omega_i L}{c} \rightarrow n, 2n, 3n, \dots$$
 (the natural frequencies of free-free rod)

What happens when  $k \rightarrow \infty$ ? In that case,  $\varphi_i$

Natural frequencies of fixed-free rod!

$$\frac{(2(i-1)+1)n}{2} = \frac{(2i-1)n}{2}, i=1,2,3,\dots$$