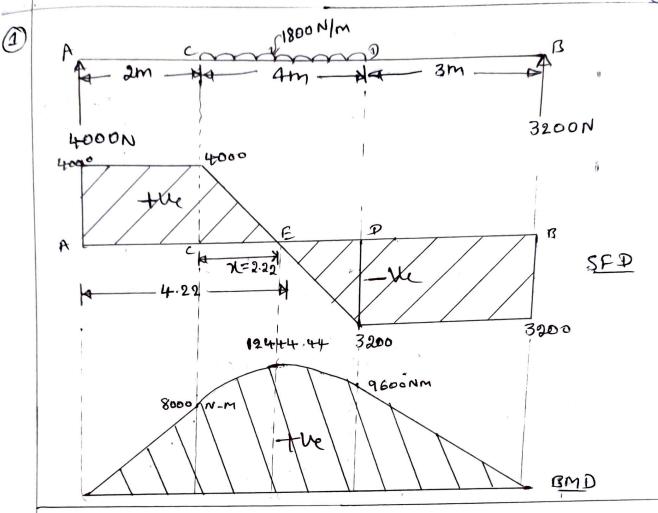
Cantilover beam with a couple acting at the free end W Fig. shows a countilever beam subjected to a couple M at the free end B. SFD: Since there is no force acting on the beam, shearing torce is zero throughout the length of the beam. BM@x-x = -M BMX: at x =0, BM@B= -M at z = E, Bm@A = - M Hence the bending moment is of constant magni tude M for the entire length of the beam. - White length - wn2 Parabolic Variation

: SFD & BMD for beams subjected to various loads:



SFD:

SF@A = 4000 SF@ before c = 4000 SF@ C = 4000 SF@ before D = 4000 - 7200

=) SFOLKED = 4000 - 1200 =) SFOLKED = - 3200

SE6 D = -3700

SF@beforeB = -3200

SF@ B = 0 1856@8:3200

BWD

BM@A f@B = 0

BM@ c = 4000X2 = 8000N-M

BM@before D= 4000 X6-7200X2

BME- = 9600N-M

BM@D = 9600 N-M

Since SFM zero at a distance x from G.

t SF@E=0

.: Shear force, SF@E = 0

=) 4000 - 1800X = 0

=) 1800x = 4000

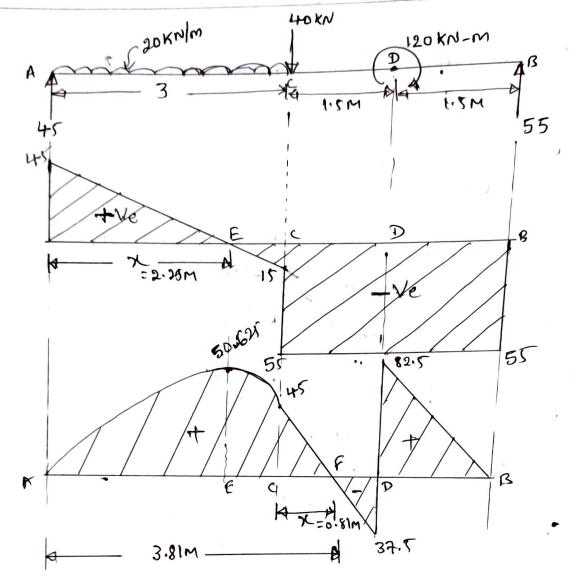
=1 X = 3.2.22 M

BM @ E is given by

= 4000×4,22-1800×222

x 3.33

= 12444 44N-M Many BM



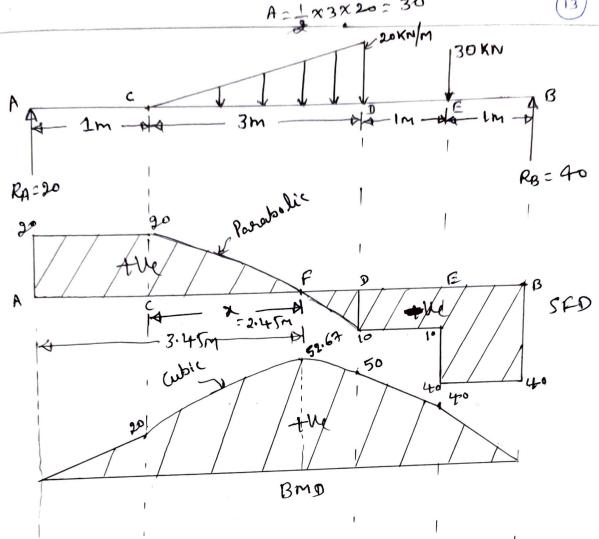
SFD:

2

SF@E(atadistance x from A) = 0

iBM @ E

NOTE: In BMD, the point Fis Called point of contraflexule where BM & zero and changes 12's sign from the to the M BM@ F = 0 => 45(3+x) - 60x(1.5+x) - 40xx 20



SFD:

=)
$$10 \frac{1}{3} = 20 \Rightarrow 12 = 20 \frac{1}{2} = 6$$

=) $10 \frac{1}{3} = 20 \Rightarrow 12 = 20 \frac{1}{2} = 6$

$$= 9 \times 248$$

$$= 90 \times 3.45 - 20 \times 2.45 = 52.67$$

BMD:

Bm@ C = 20X1 = 20

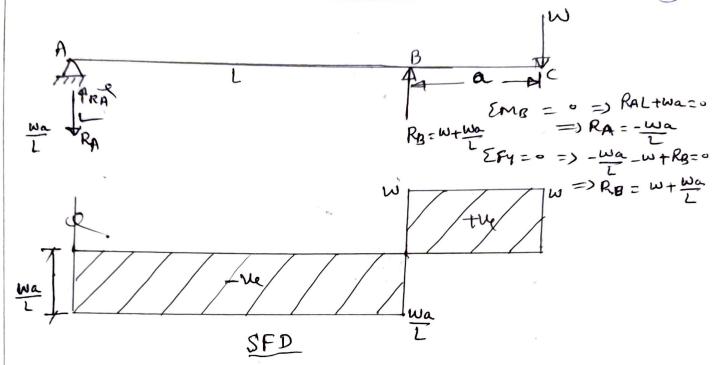
BM@beh9eD=20x4-30X1=50

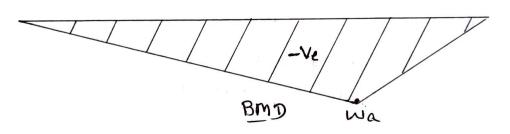
BM@ D = 50

BM @before E = 20x5-30x2=40

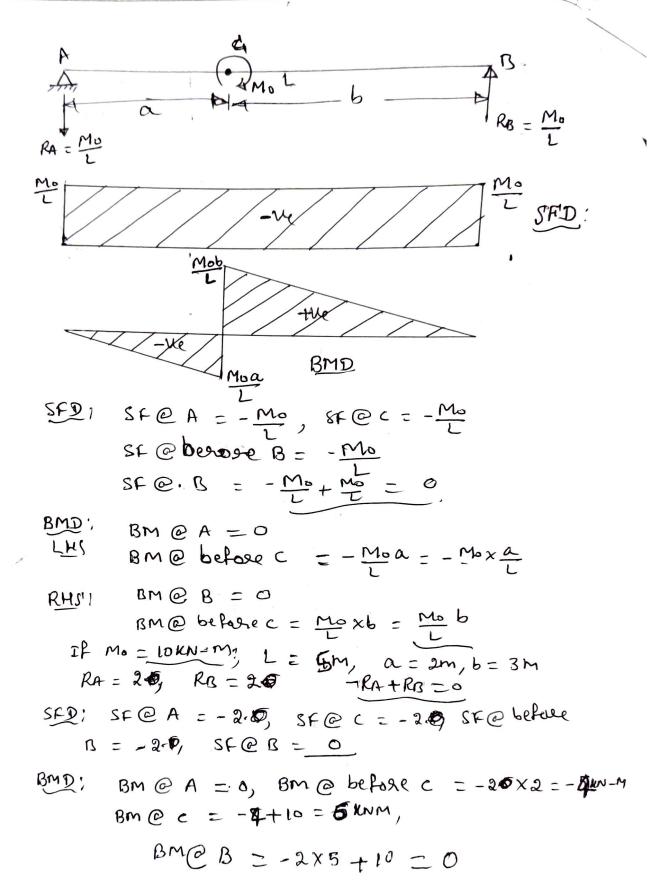
BM @B= 0

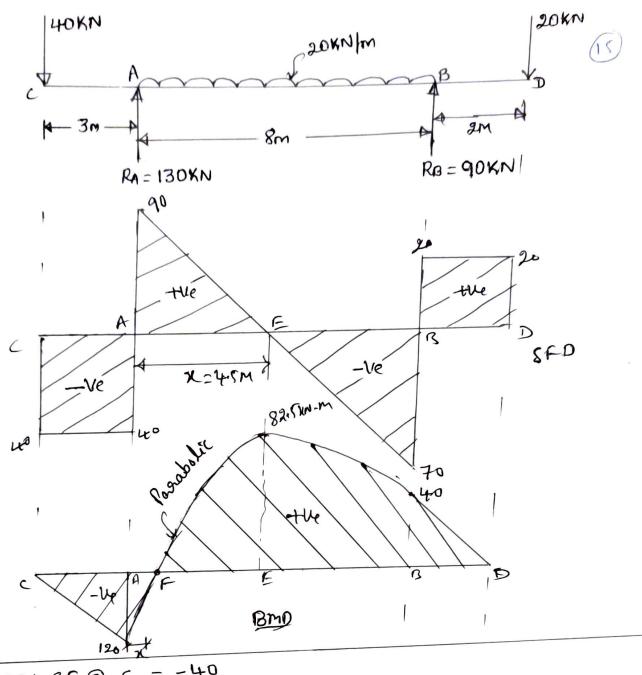
: Overhanging Beam subjected to concentrated load at:





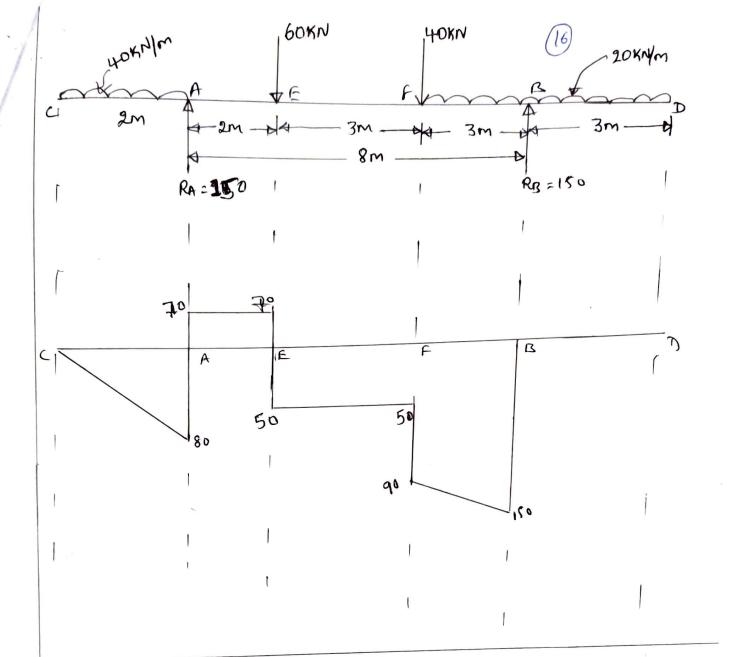
Simply supposted beam subjected to External Man @ x = a from





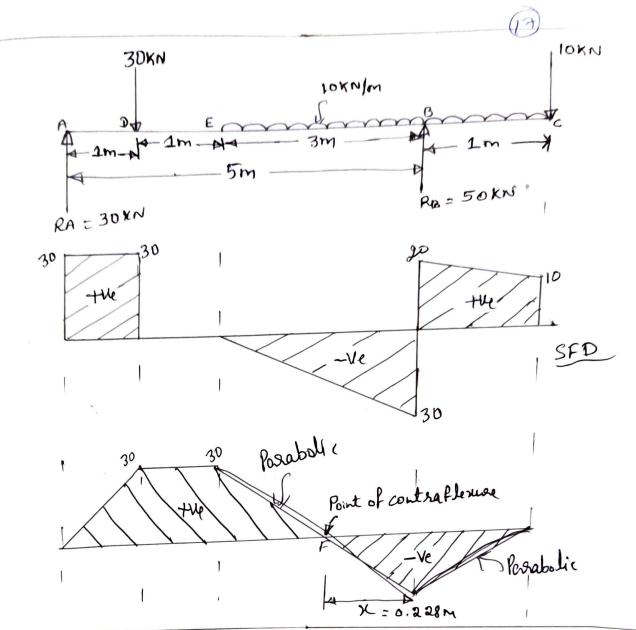
BM@A = -40x3 = -120BM@before B = -40x11 + 130x8 - 160x4 = -40BM@B = -40xH - MBM@D = 0

BM@E = -40 x 7.5+130x4.5 - 20x4.5 x 4.5 BM@E = 82.5, pt FM (alled pt of Contractionine BM@F = 0 = 1 - 40x(3+x) + 130x x - 20x2 = 0 =) -120-40x +130x - 10x2 = 0 = 2 x = 1.62m from A



Calculation of RA
$$\neq$$
 RB:

 $EMA = 0 = 3 - 80 \times 1 + 60 \times 2 + 40 \times 5 + 60 \times 6.5 - RB \times 8 + 60 \times 9.5 = 0$
 $= 3 \times RB = 1200 = 3 \times RB = 150$
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Calculation of support reactions RAJRB:

$$\sum_{\text{TMA}} = 0 \implies 30 \times 1 + 30 \times 3.5 - RB \times 5 + 10 \times 5.5 + 10 \times 6 = 0$$

$$\implies RB = 50 \text{ KN}$$

$$\Rightarrow RA = 30 - 30 + RB - 10 - 10 = 0$$

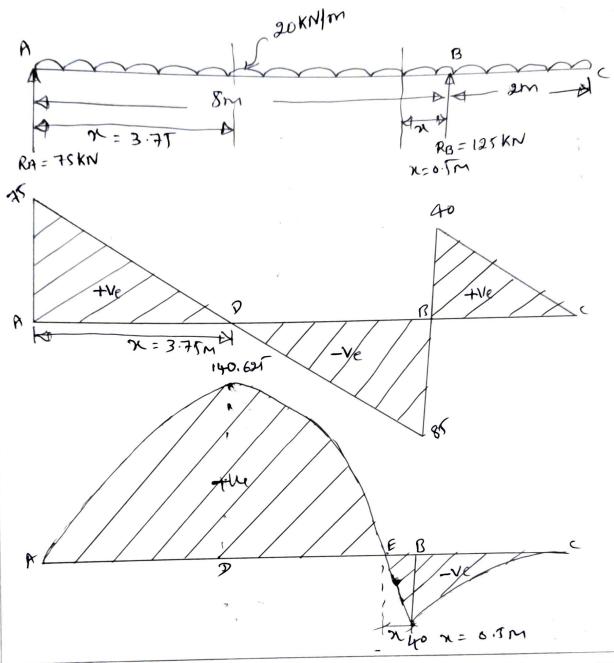
$$\Rightarrow RA = 30 + 30 + 10 + 10 - 50$$

$$\Rightarrow RA = 30 + 10 + 10 - 50$$

BMD: BM @ A $\frac{1}{2}$ = 0 BM@ D = 30×1 = 30 KN-M BM@ E = 30×2-30×1 = 60-30 = 30 KN-M BM@ before B = 30×5-30×4-30×1/5 = -15 KN-M BM@ before C = 30×6-30×5-30×2.5+50×1-10×0/7 BM@ before C = 30×6-30×5-30×2.5+50×1-10×0/7 BM@ C = 0 In BMD, It FM called point of contraflexure BM@ F = 0 $-10x(1+x) + 50x - \frac{10x^2}{2} = 0$ $-5x^2 + 45x - 10 = 0$ $ax^2 + bx + c = 0$

re a = -r, b= 4r, c = - 10

=> X = 0.228m from B end



SFD: SF@A = 75 KN, SF@ Lefoxe B = 71-160 = -85 KN

SF@B = -85+125 = 40 KN

SF@C = 40 - 40 = 0

FXLM SFD, SF@D = 0, => 75-20x =0 =) X = 3.75

BMD: BM@A = 0, BM@ LefoxeB = 75x8-160x4 = -40

BM@B = -40,

BM@C = 75x10 - 160x6+125x2-40x1 = 0

BM@OD-75x3.75-20x3.75x3.75 - 140.625km

BM @ D= 75 x 3.75 - 20 x 3.75 x 3.75 = 140.625 km-m

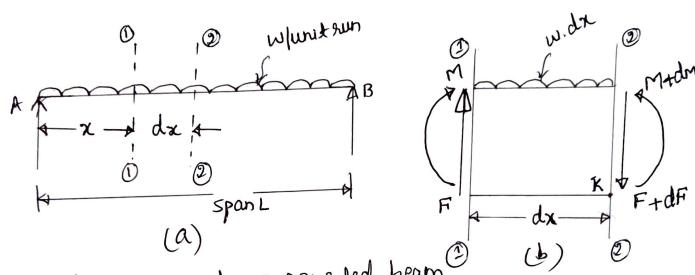
In BMD, Pt E M called point of contraflence

BM @ E = 0 = > 125 x x - 20x2 - 40(1+x) = 0

=) 125 x - 10x2 - 40 - 40x = 0 = > -10x2 + 85 x - 40 = 0.5

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Relationship between Load, shear force and Bending Moment:



Consider a simply supposed beam 3

loaded as shown in fig. about. (a) at a distance and consider an elethental length dx at a distance x from the left end A.

The face body diagram of the elemental length of the bearn along with the forces and moments acting is shown in fig.(b) about

Let Fi = shear force at section 1-1

F+dF = Shear foace at section 2-2

M = Bending moment at section 1-1 1+dm = BM at section 2-2

Total load on a beam of length dx = w.dz

For equilibrium of the elemental strip of, beam

$$\sum Fy = 0$$

$$= \sum F' - w \cdot dx - (F + dF) = 0$$

$$= \sum F' - w \cdot dx - F' - dF = 6$$

$$-w \cdot dx = dF = \sum \frac{dF'}{dx} = -w$$

=) : the above relationship shows that the sate of charge of shearfoace is equal to the load.

Now taking moments about the point K or about 2-9

$$= \int \frac{dm}{dx} = F'$$

of changes of BM is equal to shear to see

From the above, it can be seen that its case of BM semaining constant over a length firm being constant), the shear force will be zero in that Postion of the beam.