

MATLAB Paris Law for Three Point Bending

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Main The task was to find a good specimen geometry for our later experiments. For that I coded an easy MATLAB code where one defines some material parameters and can then calculate the critical crack length a_{crit} and the number of cycles for the crack to grow from a_{init} to a_{crit} with Paris Law.

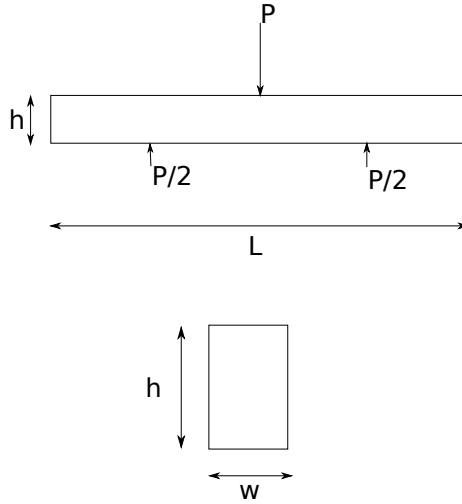


Figure 1: Setup

The code is structured as follows: First, one has to decide which length, height and width of the specimen. Then one has to specify which material is used, i.e. K_{Ic} , the initial crack length a_{init} and Paris law parameters C and m . Finally, the load P has to be defined and the β for the stress intensity factor K . Here, β can be either constant or a function of the crack length a . If β should be specified as a function, the user must pass three points of the curve into the *calcBeta* function:

1. β at $a/h = 0$
2. β at the minimum
3. β at $a/h = 0.6$

When the user has done that, he can start the script. First, σ_{max} will be calculated from

$$\sigma_{max} = \frac{M_{max}}{I} \cdot \frac{h}{2} = \frac{Pl/4}{bh^3/12} \cdot \frac{h}{2} = \frac{3}{2}P \frac{l}{bh^2}$$

With σ_{max} the critical crack length is now calculated.

$$K_{Ic} = \beta\sigma_{max}\sqrt{\pi a_{crit}} \quad (1)$$

$$\Leftrightarrow a_{crit} = \frac{1}{\pi} \frac{K_{Ic}^2}{\beta^2 \sigma_{max}^2} \quad (2)$$

With a_{crit} , we can now use Paris Law.

$$\frac{da}{dN} = C\Delta K_I^m$$

with $\Delta K_I = \beta\Delta\sigma\sqrt{\pi a}$. For sinusoidal applied force, then $\Delta\sigma = 2\sigma_{max}$ and $\Delta K_I = 2\beta\sigma_{max}\sqrt{\pi a}$. Now integration leads to

$$N_f = \int_{a_{init}}^{a_{crit}} \frac{da}{C\Delta K_I^m} = \int_{a_{init}}^{a_{crit}} \frac{da}{C(2\beta\sigma_{max}\sqrt{\pi a})^m}$$

This integral is solved by numerical integration in MATLAB in the function *paris* which uses the built-in function *integral*.

At the end, the number of cycles N_f is printed.

Appendices

Listing 1: main.m

```
1 %%
2 close all
3 clc
4 clear
5
6 %% design parameters
7 % geometry
8 ll = 100; %[mm]
9 hh = ll/8; %[mm]
10 bb = hh/2; %[mm]
11
12
13 % Paris
14 K_IC = 50; %MPa*m-0.5
15 a_init = 0.00001; %[mm]
16 C = 10-12; %[?]
17 m = 2.85; %[?]
18
19 for i=1:200
20 % load
21 P = i; %[N]
22
23 %% 1st step: Determine critical crack length
24 sigma_max = 3/2*P*ll/bb/hh2; %[MPa]
25
26 beta = @(a) calcBeta(a/hh,1.1,0.15,1.01,1.84); %[-]
27
28 a_crit = (K_IC/beta(0)/sigma_max)2/pi; %[mm]
29
30
31 %% 2nd step: Determine fatigue lifex
32
33 Nf = paris(2*sigma_max,a_init,a_crit,C,m,beta) %[-]
34 % for plotting S-N-curve
35 aaa(i)=Nf;
36 bbb(i)=sigma_max;
37 end
38 % plot
39 semilogx(aaa,bbb)
40 xlabel('N')
41 ylabel('S')
```

Listing 2: calcBeta.m

```

1  function [beta] = calcBeta(ah,beta_0,ah_min,beta_min,
    beta_06)
2
3  A = [0 0 1
4  ah_min^2 ah_min 1
5  0.6^2 0.6 1];
6  b = [beta_0;beta_min;beta_06];
7  parameters = A\b;
8
9  beta = parameters(1)*ah.^2 + parameters(2)*ah +
    parameters(3);
10
11
12 %% plot fitted function
13 % f = @(x) parameters(1)*x.^2 + parameters(2)*x +
    parameters(3);
14 % plot([0:0.01:0.6],f([0:0.01:0.6]),[0;ah_min;0.6],[
    beta_0;beta_min;beta_06])
15
16
17
18 end

```

Listing 3: paris.m

```

1  function Nf = paris(dsigma,a_init,a_end,C,m,beta)
2
3  % if beta is a function handle
4  try
5  dKinvPowm = @(a) 1./((beta(a)*dsigma.*sqrt(pi.*a)).^m;
6  Nf = 1/C*integral(dKinvPowm,a_init,a_end);
7  % if beta is constant
8  catch
9  dKinvPowm = @(a) 1./((beta*dsigma.*sqrt(pi.*a)).^m;
10 Nf = 1/C*integral(dKinvPowm,a_init,a_end);
11 end
12
13
14 end

```