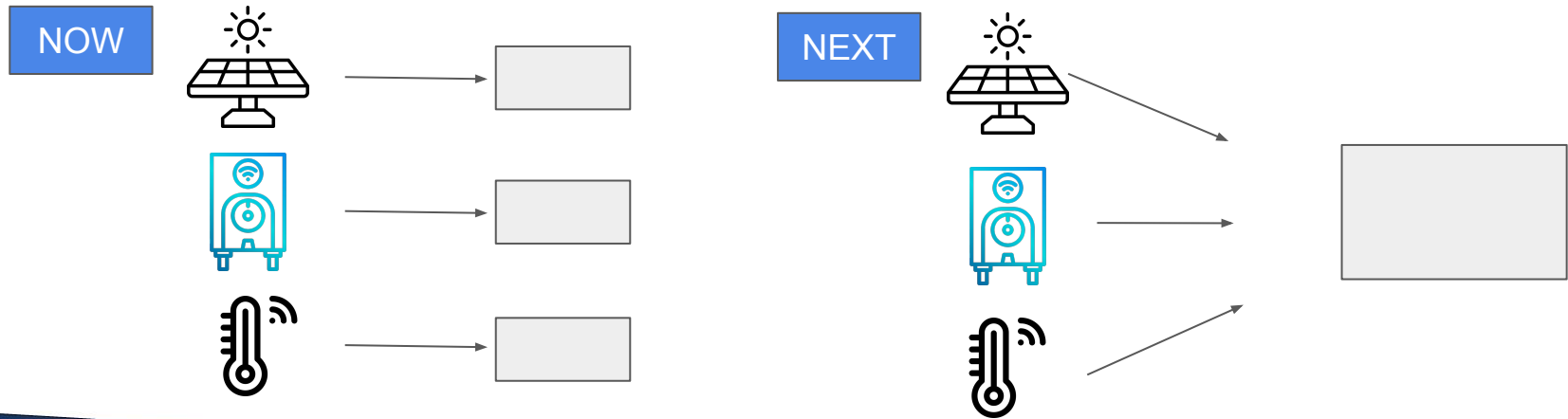


Smart Homiez: Home DER Optimization

Talia Arauzo, Chitra Dangwal, Weixin Li, Dimitris Vlachogiannis,
Karl Walter

Motivation & Background

- How can we optimize the energy consumption of in-home energy devices?
- What are the opportunities and benefits of co-optimization?



PV - Modeling

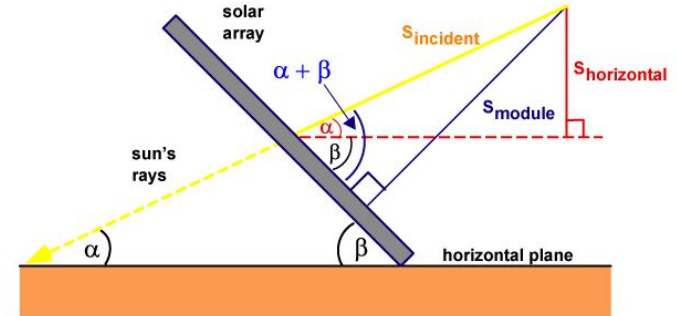
$$PV_{kw} = A_{panels} * S_{module} * \eta_{panels}$$

$$A_{panels} = n_{panels} * A_{panel}$$

$$S_{module} = GHI * \sin\left(\frac{\alpha + \beta}{\alpha}\right)$$

$$\alpha = 90 - \phi + \delta$$

$$\delta = 23.45 * \sin\left(\frac{360}{365} * (284 + d)\right)$$



PV_{kw} : power produced by solar photovoltaic (PV) array [kW]

A_{panels} : area of PV panels [m^2]

S_{module} : irradiant normal to surface of PV modules [kW / m^2]

η_{panels} : efficiency of PV

n_{panels} : number of PV panels

A_{panel} : area of single PV panel

GHI : global horizontal irradiance [kW / m^2]

α : elevation angle [radians]

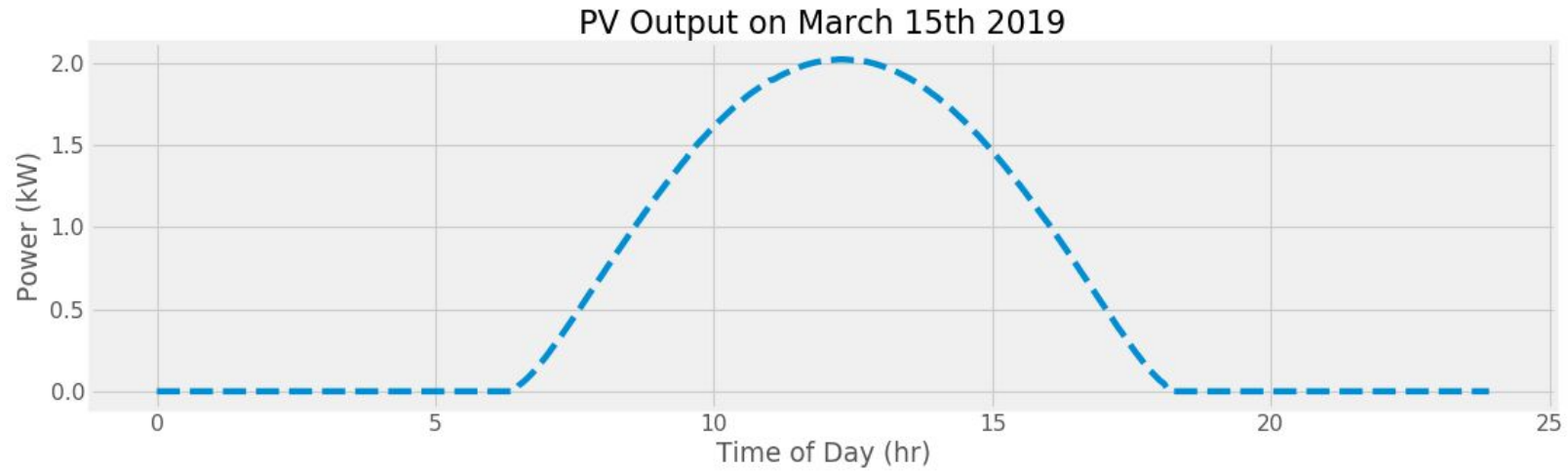
β : tilt of module [radians]

d : day of year []

ϕ : latitude of Berkeley [radians]

δ : declination angle [radians]

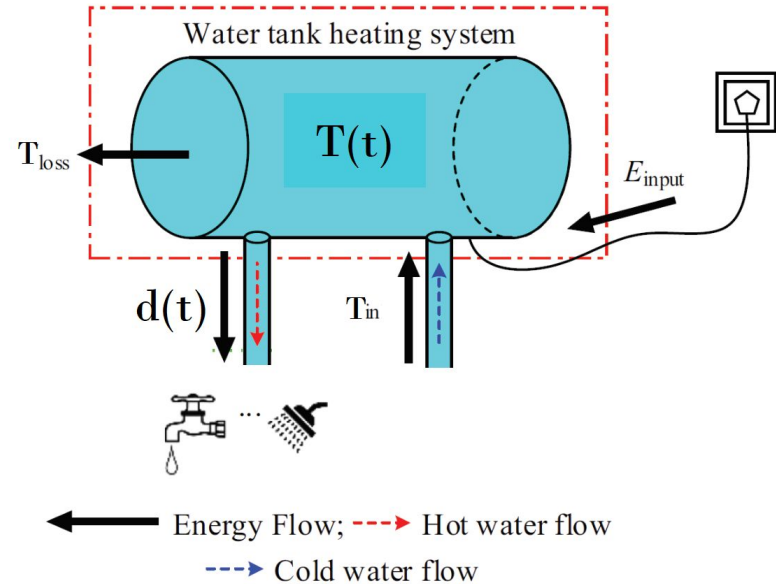
PV - Results



Water Heater - Modeling

- Model water temperature in water heater
 - Based on “Appliance Commitment for Household Load Scheduling” by Du & Lu
 - 50 gal heater
 - Thermal dynamics incorporated
 - Exogenous household hot water demands
- Simplifying assumptions
 - Output water temperature is satisfactory for user
 - Water demands are constant
 - Heat transferring process is instantaneous

$T(t)$	Temperature inside the water heater
T_{loss}	Temperature loss caused by temperature difference
T_{in}	Temperature of cold water flowing into the tank
$d(t)$	Volume of water usage
E_{input}	Energy change used to heat water



Water Heater - Optimization

Objective Function:

$$\min_{u(t), T(t)} \sum_{t=1}^n \underbrace{(p(t) \cdot Q \cdot u(t) \cdot dt)}_{\text{Cost Function}} + \underbrace{\frac{1}{2} \alpha \left[\max(0, -T(t+1) + T_{\min}) \right]^2}_{\text{Penalty Function: } T(t+1) \geq T_{\min}}$$

Constraints:

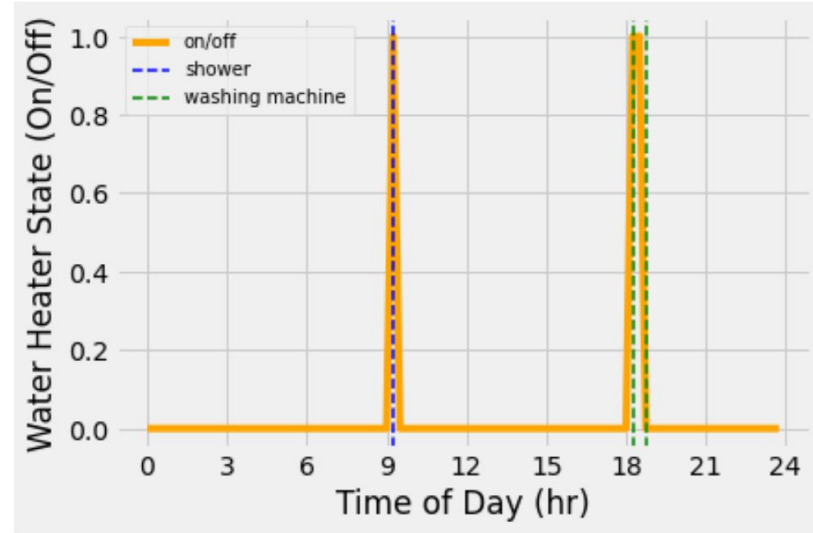
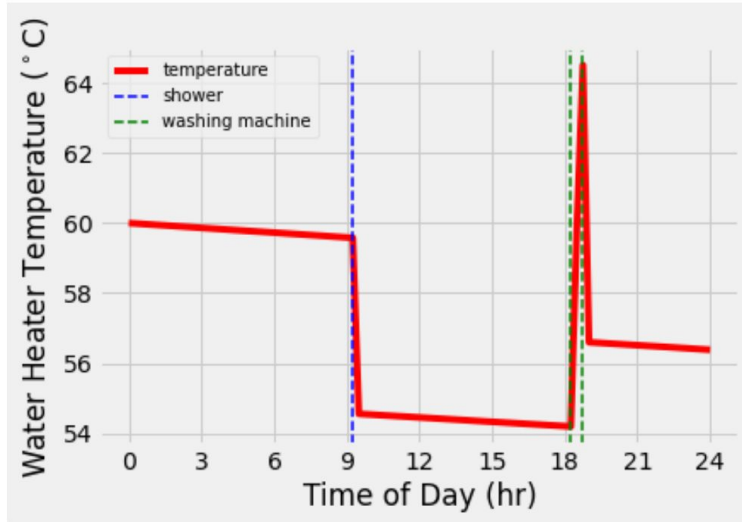
- 1) Initial temp: $T(0) = T_0$
- 2) Max temp: $T(t+1) \leq T_{\max}$
- 3) Temp + demand: $T(t+1) = \underbrace{e^{\frac{-\Delta t}{RC}} \left[\frac{M - d(t)}{M} T(t) + \frac{d(t)}{M} T_{\text{in}} \right]}_A + \underbrace{\beta Q R \left(1 - e^{\frac{-\Delta t}{RC}} \right) u(t)}_B$

Factors	
Penalty Function	$\alpha = 1$
Input Matrix	$\beta = 10,000$

Variables	
Temperature [°C]	T (t)
State [0,1]	u(t)
TOU Pricing [\$/kWh]	p(t)
Demand [gal]	d(t)

Physical Parameters	
Thermal Resistance	R = 1.52 °C/kW
Thermal Capacitance	C = 863.4 kWh/°C
Heat Capacity	Q = 4 kW
Heater Mass	M = 50 gal
Initial Temperature	T _o = 60 °C
Minimum Temperature	T _{min} = 55 °C
Maximum Temperature	T _{max} = 65 °C
Input Temperature	T _{in} = 25 °C

Water Heater - Results: \$0.91/day



Water Demands:

1. Shower (24 gal): 9-9:15 am
2. Washing machine (30 gal): 6-6:45 pm

HVAC - Modeling

Modelling Outlook:

- Thermal Model → Complexity can vary with amount of details being modelled
- Incorporation of no. of sources of heat gain and loss
- Simplified Model: Simplifying assumptions
 - 1 state thermal model chosen
 - Lowers the complexity of the in-depth physical model
 - Maintaining overall accuracy

Thermal Model

HVAC
input

$$C \frac{dT_{in}}{dt} = \frac{1}{R_t} (T_{out} - T_{in}) + \phi_{H/A} + A(1-p)\phi_{ir} + n_{ac} V_{house} \rho_{air} C_2 (T_{out} - T_{in}) / 3600$$

Heat
lost/gained
to ambient

Solar
energy
gained

Air exchanges
with the outside

Variables	
Outdoor temperature [$^{\circ}\text{C}$]	$T_{out}(t)$
Indoor temperature [$^{\circ}\text{C}$]	$T_{in}(t)$
Thermal energy flow from the sun [kW]	$\phi_{ir}(t)$
Thermal energy flow of the HVAC [kW]	$\phi_{H/A}(t) = P_{heat} Z_{heat}$
On/Off state of HVAC [0,1]	Z_{heat}

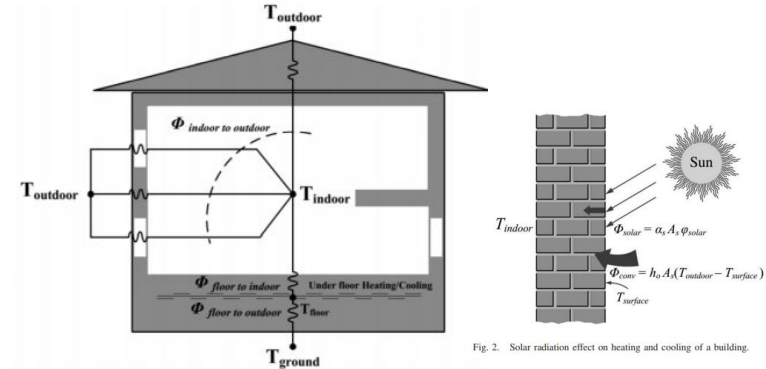


Fig. 1. Thermal modeling of a building.

Fig. 2. Solar radiation effect on heating and cooling of a building.

HVAC - Optimization

Discrete Model: $T(t_{k+1}) = A_d T(t_k) + B_d u(t_k)$

Model Input:

$$u(t_k) = \begin{bmatrix} T_{out}(t_k) \\ \phi_{H/A}(t_k) \\ \phi_{ir}(t_k) \end{bmatrix}$$

$$\left\{ \begin{aligned} A_d &= e^{A\Delta t} = e^{\left(-\frac{1}{CR} - \frac{n_{ac}V_{house}\rho_{air}C_2}{3600C}\right)\Delta t} \\ B_d &= \frac{1}{C} \begin{bmatrix} \frac{1}{R} + \frac{n_{ac}V_{house}\rho_{air}C_2}{3600} & 1 & A(1-p) \end{bmatrix} \frac{1}{\left(-\frac{1}{CR} - \frac{n_{ac}V_{house}\rho_{air}C_2}{3600C}\right)} \left(e^{\left(-\frac{1}{CR} - \frac{n_{ac}V_{house}\rho_{air}C_2}{3600C}\right)\Delta t} - 1\right) \end{aligned} \right.$$

Model Optimization: For specified heating power, optimize the variable “z(t)” i.e. ON/OFF state of HVAC $\phi_{H/A}(t_k) = P_{heat}z(t_k)$

Objective function:

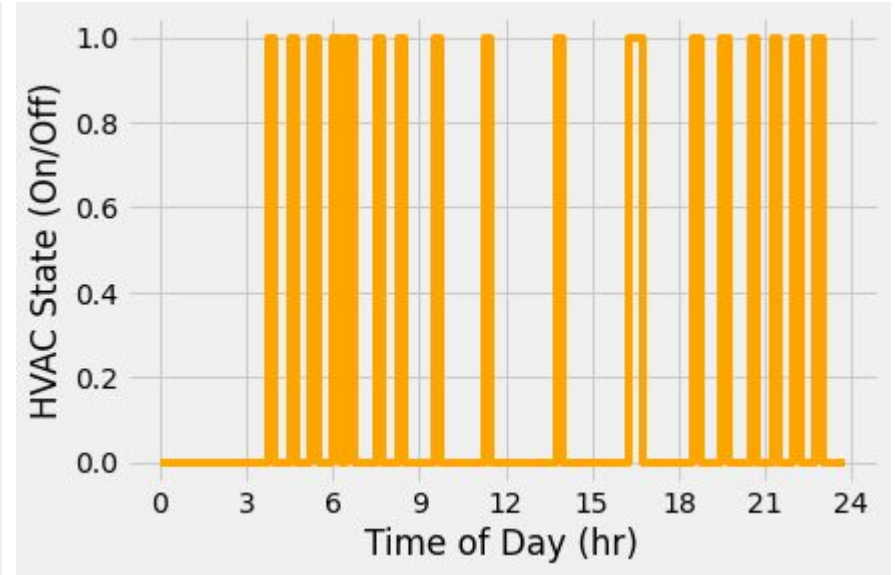
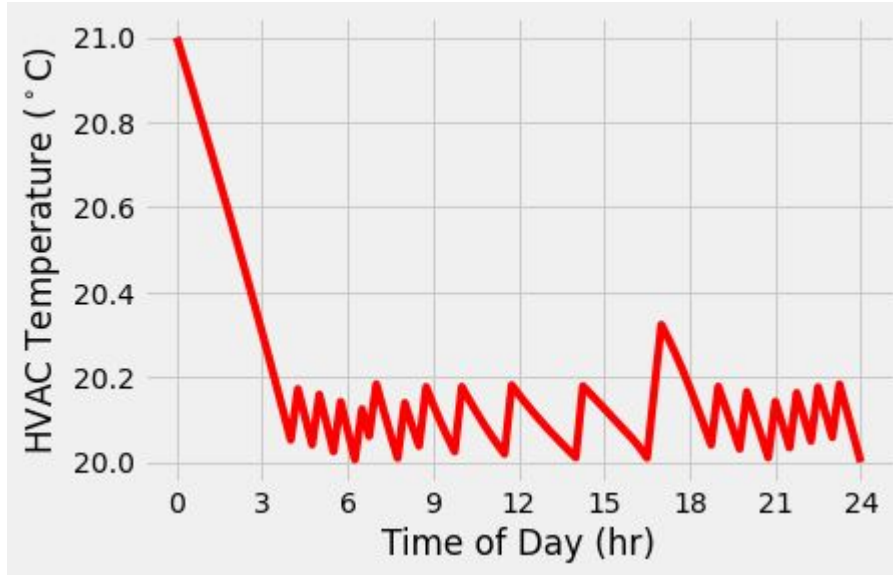
$$\min_{z(t)} \sum_{t=1}^n p(t) \cdot P_{heat} \cdot z(t)$$

Constraints :

1. Initial condition $T(0) = T_0$
2. Max temp. $T(t_k) \leq T_{max}$
3. Minimum temp. $T(t_k) \geq T_{min}$

Physical Parameters	
Capacity of the total indoor air's heat	$C=8.12$ [kWh/°C] $C_2=1214.4/\rho_{air}$ [J/(kg*°C)]
Heating power of the HVAC [kW]	$P_{heat}=60 \cdot A_{floor}/1000$
Total resistance against heat flow [°C/kW]	$R_t=8.02$
Area of the floor/window [m²]	$A_{window}=4$ $A_{floor}=100$
Volume of the house [m³]	$V_{house}=300$
Part of the irradiation of sun directly observed by window	$P=0.995$
Air density [kg/m³]	$\rho_{air}=1.2$
Number of air change of the house [1/hr]	$n_{air}=0.35$
Minimum temperature	$T_{min}=20$
Maximum temperature	$T_{max}=24$
Initial temperature	$T_0=21$

HVAC - Results: \$7.99/day



Next Steps

1

Co-Optimize: Limit consumption to solar production, i.e., $E_{WH} + E_{HVAC} \leq \beta E_{solar}$

2

Explore Robust Optimization: Investigate using uncertain demand scenarios

Thank you!

Questions?