

# Dept of Electrical and Electronic Eng. University of Cagliari

5<sup>th</sup> Workshop

**Structural Dynamical Systems: Computational Aspects** 

Capitolo (BA) – Italy

# **Sliding mode:** Basic theory and new perspectives

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#### Summary

- Switching systems
- Introductive examples
- Zeno behaviour
- Sliding modes in switching systems
- Regularization of sliding a mode trajectory
- Sliding modes in control systems
- Stability of a sliding mode control system
- Invariance of a sliding mode control system
- Recovering the uncertain dynamics
- Approximability
- Discrete time implementation
- Higher order sliding modes

### Switching systems

Switching systems are dynamical systems such that their behaviour is characterised by different dynamics in different domains

$$\dot{\mathbf{x}}(t) = f_i(\mathbf{x}(t), t, \mathbf{u}(t)), \quad \mathbf{x} \in X_i \subseteq \mathbb{R}^n, \quad \mathbf{u} \in U_i \subseteq \mathbb{R}^m, \quad t \in \mathbb{R}^+$$
$$i \in Q \subseteq \mathbb{N}$$

 $f_i$  is a smooth vector field  $f_i$ :  $\mathbf{R}^n \times \mathbf{R}^m \times \mathbf{R}^+ \rightarrow \mathbf{R}^n$ 

The state dynamics is invariant until a switch occurs

The system dynamics is represented by a differential equation with discontinuous right-hand side

### Switching systems

#### Switching between different dynamics

$$g_{i}^{sw}(\mathbf{x}(\tau_{k}), \tau_{k}, \mathbf{u}(\tau_{k}), \mathbf{v}(\tau_{k})) = 0$$

$$\downarrow \downarrow$$

$$\dot{\mathbf{x}}(\boldsymbol{\tau}_{k}^{-}) = f_{i}(\mathbf{x}(\boldsymbol{\tau}_{k}^{-}), \boldsymbol{\tau}_{k}^{-}, \mathbf{u}(\boldsymbol{\tau}_{k}^{-})), \quad \dot{\mathbf{x}}(\boldsymbol{\tau}_{k}^{+}) = f_{j}(\mathbf{x}(\boldsymbol{\tau}_{k}^{+}), \boldsymbol{\tau}_{k}^{+}, \mathbf{u}(\boldsymbol{\tau}_{k}^{+}))$$

$$\mathbf{x} \in X_i \subseteq \mathbb{R}^n$$
,  $\mathbf{u} \in U_i \subseteq \mathbb{R}^m$ ,  $\mathbf{v} \in \{0,1\}^l$   $t \in \mathbb{R}^+$   
 $i, j \in Q \subseteq \mathbb{N}$ 

The reaching of the guard  $g_i^{sw}$  cause the switching from the dynamics  $f_i$  to the dynamics  $f_i$ , according to proper rules

## Switching systems

What does it happen on the guard?  $g_i^{sw}(\mathbf{x}(\tau_k), \tau_k, \mathbf{u}(\tau_k), \mathbf{v}(\tau_k)) = 0$ 

$$\mathbf{v}(\tau_{k}) \in \phi$$

Autonomous switching

$$\mathbf{v}(\tau_k) \in \{0,1\}$$

Forced switching

$$\mathbf{x}(\tau_{_{k}}^{-}) = \mathbf{x}(\tau_{_{k}}^{+})$$

Continuous state variables

$$\mathbf{x}(\tau_{k}^{-}) \neq \mathbf{x}(\tau_{k}^{+})$$

Jumps in state variables

$$\mathbf{u}(\tau_{_{k}}^{-}) = \mathbf{u}(\tau_{_{k}}^{+})$$

Continuous control variables

$$\mathbf{u}(\tau_{_{k}}^{-})\neq\mathbf{u}(\tau_{_{k}}^{+})$$

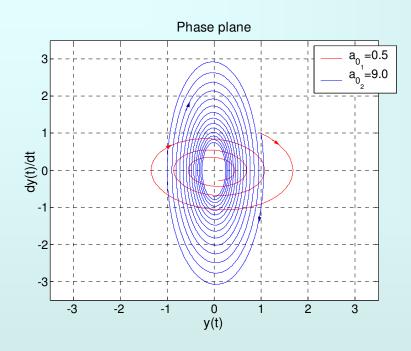
Discontinuous control variables

Most interesting...... the time evolution of the guard function!

Switching systems may behave very differently from each of the constituting ones.

$$\ddot{y}(t) + a_1 \dot{y}(t) + a_{01} y(t) = 0$$
 system 1  
 $\ddot{y}(t) + a_1 \dot{y}(t) + a_{02} y(t) = 0$  system 2  
 $0 < a_{01} < a_{02}$ 

- a<sub>1</sub>>0 the systems are both asymptotically stable
- $a_1=0$  the systems are both marginally stable
- a<sub>1</sub><0 the systems are both unstable</li>



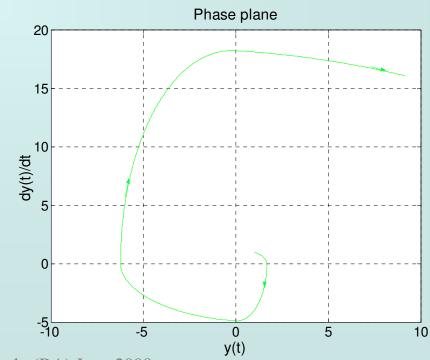
$$\ddot{y} + a_1 \dot{y} + a_0 y - \Delta a_0 |y| \operatorname{sgn}(\dot{y}) = 0$$

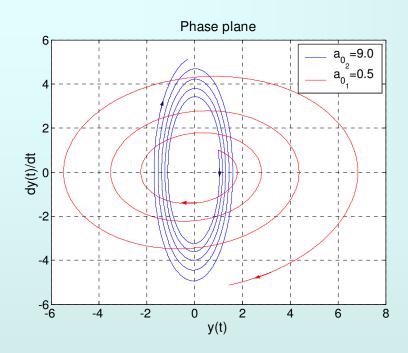
$$a_1 = 0.1$$
  $a_0 = 0.7$   $\Delta a_0 = 0.2$ 

Switched unstable dynamics

$$a_1 = 0.1$$

Both dynamics are asymptotically stable





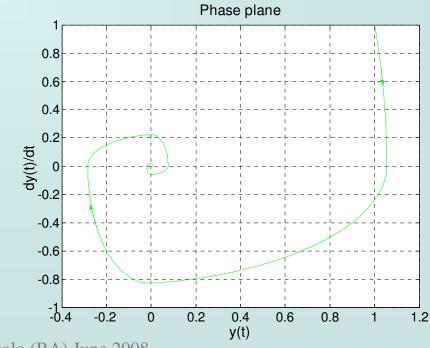
$$\ddot{y} + a_1 \dot{y} + a_0 y + \Delta a_0 |y| \operatorname{sgn}(\dot{y}) = 0$$

$$a_1 = -0.1$$
  $a_0 = 0.7$   $\Delta a_0 = 0.2$ 

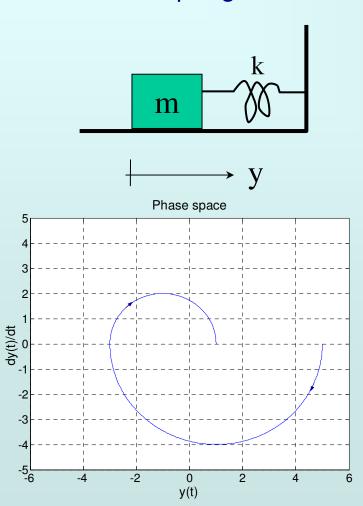
Switched asymptotically stable dynamics

$$a_1 = -0.1$$

#### Both dynamics are unstable



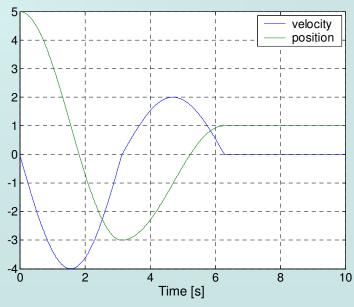
#### Mass-spring and Coulomb friction



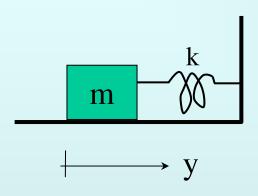
$$m\ddot{y}(t) + b\operatorname{sgn}(\dot{y}(t)) + ky(t) = 0$$

Switching time instants

$$\dot{y}(\tau_k) = 0 \to \tau_k = k\pi \sqrt{\frac{m}{k}}$$



#### Mass-spring and Coulomb friction



$$m\ddot{y}(t) + b\operatorname{sgn}(\dot{y}(t)) + ky(t) = 0$$

The guard is  $\dot{y}(t) = 0$ 

The state is continuous

The switching is autonomous

In a finite time  $\tau_{\infty}$  the mass stops!

$$\dot{y}(t) = 0$$

$$y(t) = y_{\infty} \in \left[ -\frac{b}{k}, +\frac{b}{k} \right] \qquad t \ge \tau_{\infty}$$



$$\ddot{y}(t) = -\frac{b}{m}\operatorname{sgn}(\dot{y}(t)) + \frac{k}{m}y(t) = 0 \quad t \ge \tau_{\infty}$$

#### Singular optimal control

$$\dot{\mathbf{x}} = f(\mathbf{x}) + b(\mathbf{x})u$$

$$J = \int_{0}^{T} [f_{0}(\mathbf{x}) + b_{0}(\mathbf{x})u]dt$$

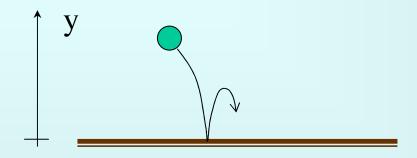
$$|u| \le 1, \quad \mathbf{x}(0) = \mathbf{x}_{0}, \quad \mathbf{x}(T) = \mathbf{x}_{T}$$

$$H = f_0(\mathbf{x}) + b_0(\mathbf{x})u + \mathbf{p}^T [f(\mathbf{x}) + b(\mathbf{x})u]$$
$$\dot{\mathbf{p}} = -\left[\frac{\partial H}{\partial \mathbf{x}}\right]^T$$
$$u = -\operatorname{sgn}(b_0(\mathbf{x}) + \mathbf{p}^T b(\mathbf{x}))$$

$$b_0(\mathbf{x}) + \mathbf{p}^T b(\mathbf{x}) = 0 \quad t \in [t_1, t_2]$$

u switches at infinite frequencyu is defined looking at the higher derivatives of the guard

#### The bouncing ball



$$\ddot{y}(t) = -g \quad y \ge 0$$

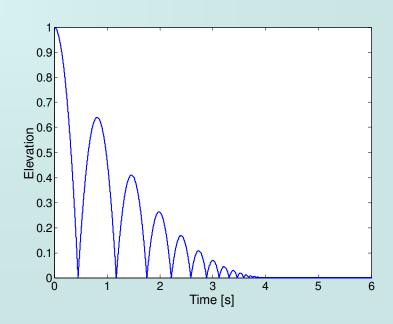
$$\tau_i : y(\tau_i) = 0 \quad (i = 1, 2, ...)$$

$$\dot{y}(\tau_i^+) = -\alpha \dot{y}(\tau_i^-) \quad \alpha \in [0, 1]$$

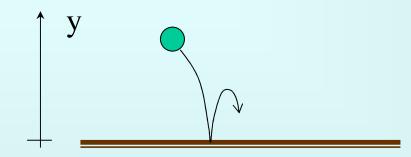
$$\tau_{k+1}^{-} - \tau_{k}^{+} = \frac{2}{g} \alpha^{k} \sqrt{\dot{y}_{0}^{2} + 2gy_{0}}$$

$$\dot{y}_{k} = \dot{y}(\tau_{k}^{+}) = \alpha^{k} \sqrt{\dot{y}_{0}^{2} + 2gy_{0}}$$

$$\tau_{\infty} = \frac{1}{g} \dot{y}_{0} + \frac{1}{g} \sqrt{\dot{y}_{0}^{2} + 2gy_{0}} \frac{1 + \alpha}{1 - \alpha}$$



#### The bouncing ball



with abuse of notation

$$\ddot{y}(t) = -g - (1 + \alpha)\dot{y}(t) \cdot \delta(y) \quad y \ge 0$$

$$\alpha \in [0,1]$$

 $\delta(\bullet)$  is the Dirac's function

In a finite time  $\tau_{\infty}$  the mass stops!

$$y(t) = 0$$

$$\dot{y}(t) = 0$$

$$t \ge \tau_{\infty}$$



$$\ddot{y}(t) = -g - (1 + \alpha)\dot{y}(t) \cdot \delta(y) = 0 \quad t \ge \tau_{\infty}$$



#### Zeno behaviour

The switching behaviour of a hybrid system can be described by its execution set

$$\chi^{\mathsf{H}} = \{T, In, Ed\}$$

 $T = \{\tau_i\}_{i \in \mathbb{N}}$ : set of switching/jump time instants  $In = \{\mathbf{x}_i\}_{i \in \mathbb{N}} \mathbf{x}_i \subseteq D$ : set of initial states sequence  $Ed = \{\eta_i\}_{i \in \mathbb{N}} \eta_i = (i,j) \subseteq Q \times Q$ : set of edge sequence

The execution  $\chi^H$  of the hybrid system H can be constituted by finite or infinite elements.

Executions with infinite number of elements may be due to the **Zeno** phenomenon

#### Zeno behaviour

The Zeno phenomenon appears when the execution  $\chi^{H}\,$  of the hybrid system is such that

$$\lim_{i\to\infty}\tau_i=\sum_{i=0}^{\infty}\left(\tau_{i+1}-\tau_i\right)=\tau_{\infty}<\infty$$

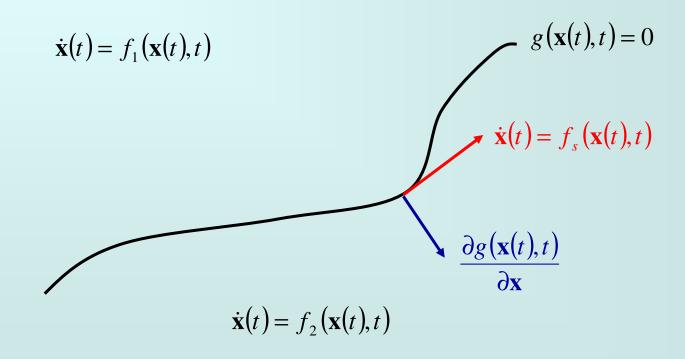
 $au_{\infty}$  (**Zeno time**) is a right accumulation point for the time instants sequence

$$(\tau_{i+1} - \tau_i) \underset{i \to \infty}{\longrightarrow} 0$$
 The switching frequency tends to be infinite

In a Zeno condition the system evolves along a guard

$$\frac{\partial g(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot \dot{\mathbf{x}}(t) = 0 \quad \forall t \ge \tau_{\infty}$$

The Zeno phenomenon is mainly related to the switching frequency on a guard, but previous relationship shows the relation between the guard and the system dynamics



The motion of the system on a discontinuity surface is called sliding mode

- ✓ Sliding modes are Zeno behaviours in switching systems
- ✓ The system is constrained onto a surface in the state space, the sliding surface
- ✓ When the system is constrained on the sliding surface, the system modes differ from those of the original systems
- ✓ The system is invariant when constrained on the sliding surface
- ✓ Any system on the sliding surface behaves the same way

$$\ddot{y} - a_1 \dot{y} + a_0 y = 0$$

$$a_1 > 0$$
  $a_0 = \pm \alpha$ 

$$a_0 = \alpha$$
  $a_0 = -\alpha$ 

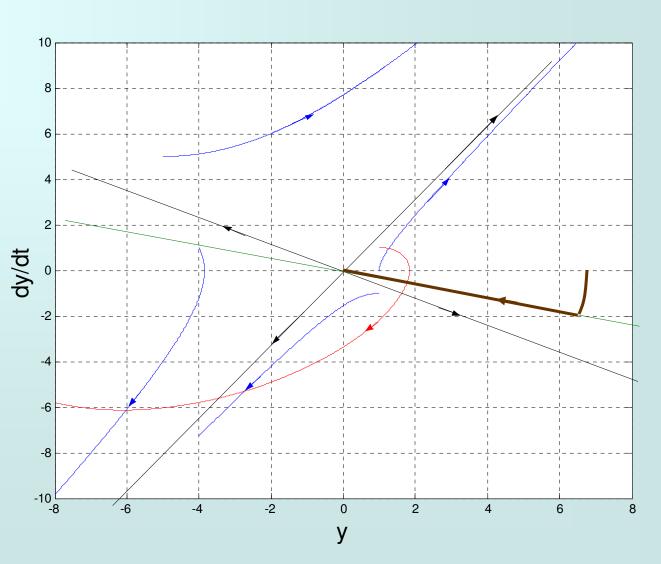
 $\alpha > 0$ 

$$\alpha > 0 \qquad \qquad \forall \sigma$$

$$\ddot{y} - a_1 \dot{y} + a_0 \operatorname{sgn}(y\sigma) = 0 \qquad \forall \sigma$$

 $\sigma = \dot{y} + cy$ 

$$a_1 = 1$$
,  $a_2 = 1$ ,  $c = 0.2$ ,

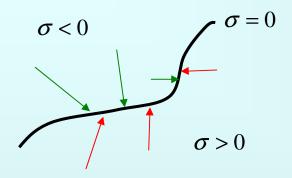


When considering switched systems, the system dynamics can be represented by a discontinuous right-hand side differential equation

$$\dot{\mathbf{x}}(t) = \begin{cases} f_1(\mathbf{x}(t), t) & \sigma(\mathbf{x}(t), t) < 0 \\ f_2(\mathbf{x}(t), t) & \sigma(\mathbf{x}(t), t) > 0 \end{cases}$$

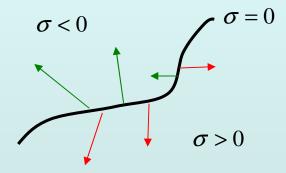
 $\sigma(\mathbf{x}(t),t)=0$  Represents the boundary between two distinct regions of the state space, possibly time varying

The behaviour of the system on/across the guard  $\sigma$ , defining the two distinct regions of the state space, depends on how the dynamics  $f_1$  and  $f_2$  are related to the switching surface



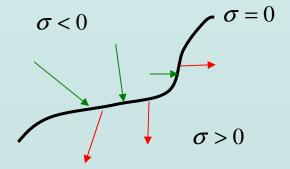
$$\begin{cases}
\frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_2(\mathbf{x}(t),t) < 0 \\
\frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_1(\mathbf{x}(t),t) > 0
\end{cases}$$

attractive switching surface



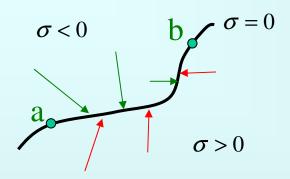
$$\begin{cases} \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_2(\mathbf{x}(t),t) > 0 \\ \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_1(\mathbf{x}(t),t) < 0 \end{cases}$$

repulsive switching surface



$$\begin{cases}
\frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_2(\mathbf{x}(t),t) < 0 \\
\frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_1(\mathbf{x}(t),t) > 0
\end{cases}$$

across switching surface



$$\begin{cases} \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_2(\mathbf{x}(t),t) < 0 \\ \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} \cdot f_1(\mathbf{x}(t),t) > 0 \end{cases}$$

attractive switching surface

A Sliding Mode appears in the segment [a, b] of the switching surface if

$$\begin{cases}
\lim_{\sigma \to 0^{+}} \frac{\partial \sigma(\mathbf{x}(t), t)}{\partial \mathbf{x}} \cdot f_{2}(\mathbf{x}(t), t) < 0 \\
\lim_{\sigma \to 0^{-}} \frac{\partial \sigma(\mathbf{x}(t), t)}{\partial \mathbf{x}} \cdot f_{1}(\mathbf{x}(t), t) > 0
\end{cases}$$

$$\forall \mathbf{x} \in [\mathbf{a}, \mathbf{b}] \subseteq \{\mathbf{x} : \sigma(\mathbf{x}(t), t) = 0\}$$

The Sliding Mode is stable in the segment [a, b] of the switching surface

#### Considering the switched dynamics

$$\dot{\mathbf{x}}(t) = \begin{cases} f_1(\mathbf{x}(t), t) & \sigma(\mathbf{x}(t), t) < 0 \\ f_s(\mathbf{x}(t), t) & \sigma(\mathbf{x}(t), t) = 0 \\ f_2(\mathbf{x}(t), t) & \sigma(\mathbf{x}(t), t) > 0 \end{cases}$$

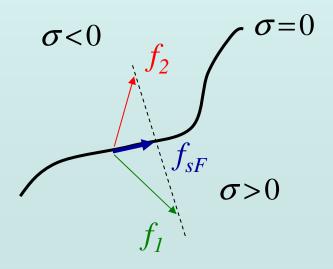
Regularization of the sliding mode implies to find the continuous vector function  $f_s$  such that the state trajectory remains on the switching surface

- √ Filippov's continuation method
- ✓ Equivalent dynamics method ("equivalent control")

#### Filippov's continuation method

$$f_{sF}(\mathbf{x}(t),t) = \alpha \ f_2(\mathbf{x}(t),t) + (1-\alpha)f_1(\mathbf{x}(t),t),$$
  
$$\alpha \in [0,1]: \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} f_{sF}(\mathbf{x}(t),t) = 0 \quad \forall$$

The dynamics on the sliding surface is defined by a convex combination of the vector fields defined in the two separated regions of the state space



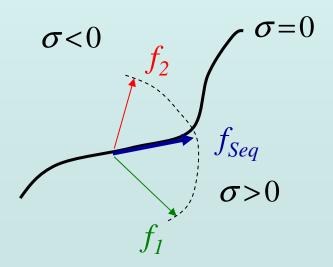
$$\alpha = \frac{grad(\sigma) \cdot f_1}{grad(\sigma) \cdot (f_1 - f_2)},$$

$$1 - \alpha = \frac{grad(\sigma) \cdot f_2}{grad(\sigma) \cdot (f_1 - f_2)},$$

#### Equivalent dynamics method

It is used when the discontinuity is due to switching of an independent variable, the "control", u(t)

$$\dot{\mathbf{x}}(t) = \begin{cases} f(\mathbf{x}(t), t, \mathbf{u}_1(t)) & \sigma(\mathbf{x}(t), t) < 0 \\ f(\mathbf{x}(t), t, \mathbf{u}_{eq}(t)) & \sigma(\mathbf{x}(t), t) = 0 \\ f(\mathbf{x}(t), t, \mathbf{u}_2(t)) & \sigma(\mathbf{x}(t), t) > 0 \end{cases}$$



$$f_{seq}(\mathbf{x}(t),t,\mathbf{u}(t)) = f(\mathbf{x}(t),t,\mathbf{u}_{eq}(t))$$

$$\mathbf{u}_{eq}: \frac{\partial \sigma(\mathbf{x}(t),t)}{\partial \mathbf{x}} f_{seq}(\mathbf{x}(t),t,\mathbf{u}(t)) = 0$$

Filippov's continuation method and the equivalent control method can give different solutions in nonlinear systems



Not uniqueness problems in finding the solution of the sliding mode dynamics



Presence of more than a single switching surface

In real systems such motions appear in representing constrained systems, or in control systems in which a variable is forced to be zero



In most cases the uniqueness problem does not arise

#### Sliding modes in control systems

Usually sliding mode control systems consider dynamics with affine control

$$\dot{\mathbf{x}} = f(\mathbf{x}, t) + B(\mathbf{x}, t)\mathbf{u}$$

$$\mathbf{\sigma}(\mathbf{x}, t) = 0$$

$$\mathbf{x} \in R^{n} \quad \mathbf{u} \in R^{m} \quad \mathbf{\sigma} \in R^{m}$$

The control vector is such that

$$u_{i}(\mathbf{x},t) = \begin{cases} u_{i}^{+}(\mathbf{x},t) & if \quad \sigma_{i}(\mathbf{x},t) > 0 \\ u_{i}^{-}(\mathbf{x},t) & if \quad \sigma_{i}(\mathbf{x},t) < 0 \end{cases} \qquad i = 1,2,...,m$$

the sliding manifold is constituted by the intersection of m switching surfaces, and uniqueness problems can arise

Local stability of a sliding mode on the domain

$$S(t) = {\sigma: \sigma_i = 0; i = 1, 2, ..., m}$$

can be stated by the following Lyapunov-like theorem

The subspace S(t) is a sliding domain if a function  $V(\sigma, x, t)$  exists in a domain  $\Omega$  of the space  $\{\sigma_1, \sigma_2, ..., \sigma_m\}$  containing the origin such that:

- $V(\sigma, x, t)$  is positive definite with respect to  $\sigma$ , for all x and any t
- On the sphere  $||\sigma||=R$ , for all x and any t, the function  $V(\sigma,x,t)$  remains bounded
- The total time derivative of function  $V(\sigma, x, t)$

$$\dot{V} = \left(\frac{\partial V}{\partial \mathbf{\sigma}} \frac{\partial \mathbf{\sigma}}{\partial \mathbf{x}} + \frac{\partial V}{\partial \mathbf{x}}\right) (f + B\mathbf{u}) + \frac{\partial V}{\partial t}$$

is negative everywhere except on the discontinuity surfaces where it is not defined

• On the sphere  $||\sigma|| = R$ , for all x and any t, the total time derivative of  $V(\sigma, x, t)$  is upper bounded by a negative constant

The sufficient conditions for stability of the sliding mode are not easy to verify in general but some specific results can be given

$$\dot{\mathbf{x}} = f(\mathbf{x}, t) + B(\mathbf{x}, t)\mathbf{u}$$
  
 $\mathbf{\sigma}(\mathbf{x}, t) = 0$   
 $\mathbf{x} \in R^n \quad \mathbf{u} \in R^m \quad \mathbf{\sigma} \in R^m$ 

$$V(\mathbf{\sigma}) = \frac{1}{2}\mathbf{\sigma}^{T} \left( \frac{\partial \mathbf{\sigma}}{\partial \mathbf{x}} \cdot B \right)^{-1} \mathbf{\sigma} = \frac{1}{2}\mathbf{\sigma}^{T} (C \cdot B)^{-1} \mathbf{\sigma} \implies \dot{V}(\mathbf{\sigma}) = \mathbf{\sigma}^{T} (C \cdot B)^{-1} C(\mathbf{x}, t) \cdot (f(\mathbf{x}, t) + B(\mathbf{x}, t)\mathbf{u})$$

$$u_i = -\alpha \operatorname{sgn}(\sigma_i); \quad \alpha b_i > |f_i| + k \quad \forall i \quad \Rightarrow \quad \operatorname{sgn}(f + \alpha B \mathbf{u}) = \operatorname{sgn}(\mathbf{u}) = -\operatorname{sgn}(\sigma)$$



$$\dot{V}(\mathbf{\sigma}) = \mathbf{\sigma}^T (CB)^{-1} C \cdot (f - B \operatorname{sgn}(\mathbf{\sigma})) \le -k\mathbf{\sigma}^T \operatorname{sgn}(\mathbf{\sigma}) = -\beta \sqrt{V(\mathbf{\sigma})}$$

#### Convergence to the sliding mode in finite time

SISO systems

$$\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}, u(t), t)$$

$$\mathbf{x} \in R^{n} \quad \sigma \in R \quad u \in R$$

$$\sigma = \mathbf{h}(\mathbf{x}, t)$$

r is the relative-degree

$$y^{(r)} = \varphi(\mathbf{x}, u, t); \quad \frac{\partial y^{(i)}}{\partial u} = 0 \quad i = 1, 2, ..., r - 1 < n$$

The control can stabilise the output vector  $\mathbf{y}$ , but the internal dynamics  $\mathbf{w}$  can be unstable for  $\mathbf{y} = 0$  (zero-dynamics)

$$\mathbf{y} = \left[\sigma, \dot{\sigma}, \ddot{\sigma}, \dots, \sigma^{(r-1)}\right]^{T},$$

$$\sigma = \varphi'(\mathbf{y}, \mathbf{w}, u, t); \qquad \mathbf{w} = \left[w_{1}, w_{2}, \dots, w_{n-r}\right]^{T},$$

$$\dot{\mathbf{w}} = \psi(\mathbf{y}, \mathbf{w}, t) \qquad \mathbf{x} = \Theta(\mathbf{y}, \mathbf{w}),$$

$$\varphi' = \varphi(\Theta(\mathbf{y}, \mathbf{w}), u, t)$$

#### Example

$$\dot{x}_1 = x_2$$

$$\dot{x}_2 = -x_1 - x_2 |x_2| + u$$

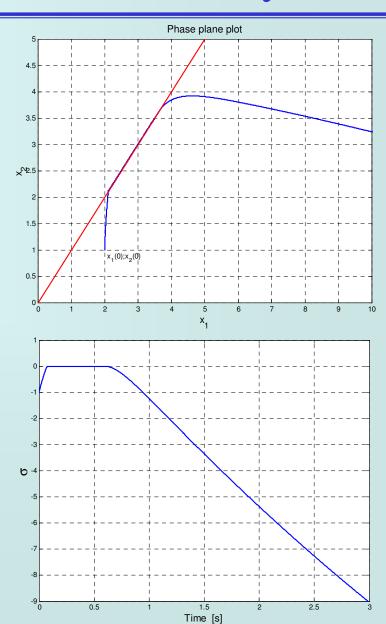
$$\sigma = x_2 - x_1$$

$$\dot{\sigma} = -x_2(1+|x_2|)-x_1+u$$

$$x_1(0) = 2; x_2(0) = 1$$

$$u = -20 \operatorname{sgn} \sigma$$

The unstable zero-dynamics causes the loss of the sliding mode behaviour



The system is invariant when constrained on the sliding manifold  $\sigma$ 

$$\dot{x}_i = x_{i+1} \quad i = 1, 2, \dots n-1$$
  
$$\dot{x}_n = f(\mathbf{x}, t) + b(\mathbf{x}, t)u$$

 $|f(\mathbf{x},t)| \le F(\mathbf{x}), \qquad 0 < b_m(\mathbf{x}) \le b(\mathbf{x},t)$ 

The system is uncertain with known bounds

 $c_{\rm i}$  are chosen such that the corresponding polynomial is Hurwitz

Finite time convergence to the sliding manifold is assured

$$\sigma = x_n + \sum_{i=1}^{n-1} c_i x_i$$

$$\dot{\sigma} = f(\mathbf{x}, t) + b(\mathbf{x}, t) u + \sum_{i=1}^{n-1} c_i x_{i+1}$$

$$u = -\operatorname{sgn} \frac{F(\mathbf{x}) + \left| \sum_{i=1}^{n-1} c_i x_{i+1} \right| + k^2}{b_m(\mathbf{x})} \implies \sigma \dot{\sigma} \leq -k^2 \sigma$$

The system is invariant when constrained on the sliding manifold  $\sigma$ 

$$\dot{x}_{i} = x_{i+1} \quad i = 1, 2, \dots n-2$$

$$\dot{x}_{n-1} = -\sum_{i=1}^{n-1} c_{i} x_{i} + \sigma$$

$$x_{n} = -\sum_{i=1}^{n-1} c_{i} x_{i} + \sigma$$

The system behaves as a reduced order system with prescribed eigenvalues

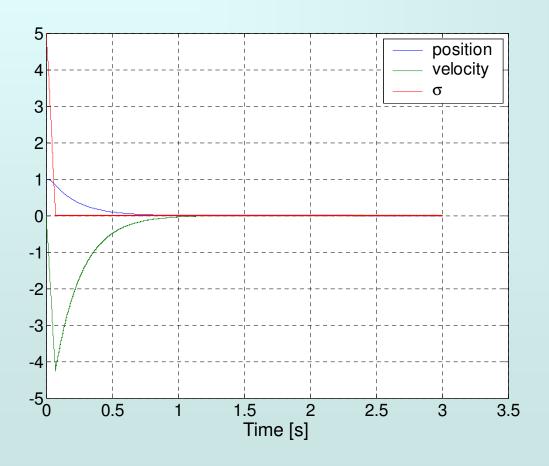
Matching uncertainties, included in the uncertain function f, are completely rejected

In the sliding mode it is not possible to "recover" the original system dynamics (*semi-group property*)

Take care of unstable zero-dynamics

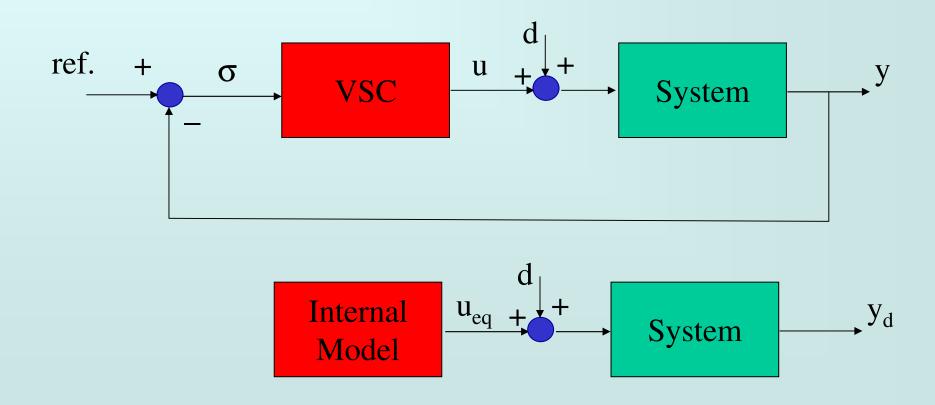
$$m(t)\ddot{y} + (b_1 + b_2|\dot{y}|)\dot{y} + b_3\operatorname{sgn}(\dot{y}) + (k_1 + k_3y^2)y = u - \sin(\pi t)$$

$$\sigma = \dot{y} + cy$$



$$u = -U \operatorname{sgn}(\sigma)$$

The invariance property during the sliding mode means that the "Internal Model Principle" is fulfilled



The controlled system dynamics belongs to a differential inclusion

$$\dot{\mathbf{x}} \in \mathsf{F} = f(\mathbf{x}, t) + b(\mathbf{x}, t) [-U, +U]$$

The sliding variable  $\sigma$  can be considered as a performance index to be nullified to find the "right" solution

$$\dot{\mathbf{x}}^* = f(\mathbf{x}, t) + b(\mathbf{x}, t) u_{eq} \in \mathsf{F}$$

#### Recovering the uncertain dynamics

$$\tau u_{av} + u_{av} = u$$

If  $u_{eq}$  is bounded with its time derivative then

$$\lim_{\substack{\tau \to 0 \\ \frac{\Delta}{\tau} \to 0}} u_{av} = u_{eq} \qquad |\sigma| \le \Delta$$

The cut-off frequency of the low-pass filter must be

- Greater than the bandwidth of the equivalent control
- Lower than the real switching frequency

In practice only an estimate of  $u_{eq}$  can be evaluated

## Recovering the uncertain dynamics

The *equivalent control* is the control signal that assures  $\dot{\sigma} \equiv 0$ 

$$\sigma = x_n + \sum_{i=1}^{n-1} c_i x_i$$

$$\dot{\sigma} = f(\mathbf{x}, t) + b(\mathbf{x}, t) u + \sum_{i=1}^{n-1} c_i x_{i+1}$$

$$u_{eq} = -\frac{f(\mathbf{x}, t) + \sum_{i=1}^{n-1} c_i x_{i+1}}{b(\mathbf{x}, t)} \implies \dot{\sigma} \equiv 0$$

The *equivalent control* contains the information on the uncertain dynamics

$$U(j\omega) = U_{eq}(j\omega) + \overline{U}(j\omega)\Big|_{\omega=\infty}$$

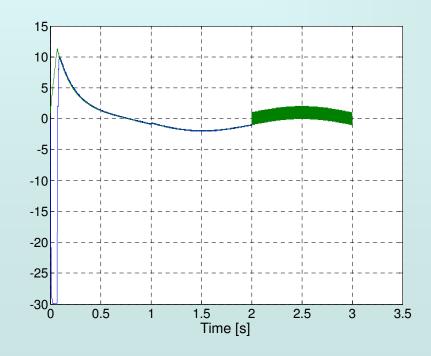
The equivalent control can be estimated by a low-pass filter

$$\tau u_{av} + u_{av} = u$$

## Recovering the uncertain dynamics

$$m(t)\ddot{y} + (b_1 + b_2|\dot{y}|)\dot{y} + b_3\operatorname{sgn}(\dot{y}) + (k_1 + k_3y^2)y = u - \sin(\pi t)$$
  
$$\sigma = \dot{y} + cy$$

$$u_{eq} = +(b_1 + b_2|\dot{y}|)\dot{y} + b_3 \operatorname{sgn}(\dot{y}) + (k_1 + k_3 y^2)y + \sin(\pi t) + cm(t)\dot{y}$$



# **Approximability**

The dynamics of the original system can be reduced to

$$\dot{x}_{i} = x_{i+1} \quad i = 1, 2, \dots n - 2$$

$$\dot{x}_{n-1} = -\sum_{i=1}^{n-1} c_{i} x_{i} + \sigma$$

$$x_{n} = -\sum_{i=1}^{n-1} c_{i} x_{i} + \sigma$$

In the case of switching errors – the switching frequency is  $f_s \le \infty$  – the real trajectory  $\mathbf{x}(t)$  is near to the ideal one  $\mathbf{x}^*(t)$  and

$$\|\mathbf{x}(t) - \mathbf{x}^*(t)\| \xrightarrow{f_{s \to \infty}} 0$$

Assuming that the dynamics of the original system is bounded

$$\dot{\mathbf{x}} = f(\mathbf{x}, t) + b(\mathbf{x}, t) [-U, +U]$$
$$\| f(\mathbf{x}, t) + b(\mathbf{x}, t) u(\mathbf{x}, t) \| \le M + N \|\mathbf{x}\|$$

# Approximability

Starting from a vicinity of the sliding surface

$$\sigma = C(\mathbf{x}) \cdot \mathbf{x}$$
$$\|\mathbf{x}_0\| \le \Delta$$

By the Bellman-Gronwall lemma

$$\|\mathbf{x}(t)\| \le M \|\mathbf{x}_0\| + MT + \int_0^T N \|\mathbf{x}\| dt$$

$$\|\mathbf{x}(t) - \mathbf{x}^*(t)\| \le S\Delta + L \int_0^T \|\mathbf{x}(t) - \mathbf{x}^*(t)\| dt \le H\Delta$$

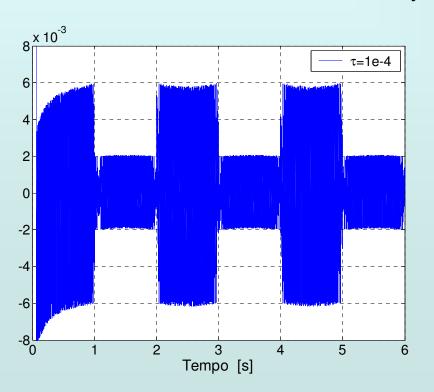
If  $\tau$  is the equivalent delay of the switching

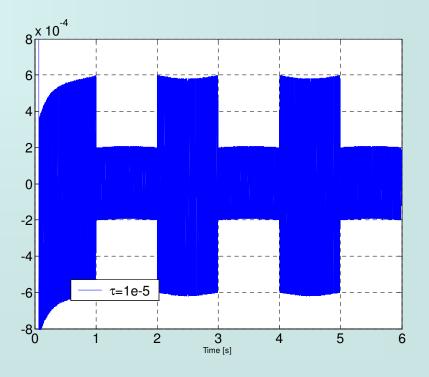
$$\begin{aligned} & \|\dot{\sigma}\| \leq \Sigma \\ & \|\sigma\| \leq & \|\sigma_0\| + \Sigma\tau \\ & \Delta = D\tau \end{aligned}$$

# Approximability

The most common cause of delay switching is the digital implementation of the controller

$$m(t)\ddot{y} + (b_1 + b_2|\dot{y}|)y + (k_1 + k_3 y^2)y = u - \sin(\pi t)$$
  
$$\sigma = \dot{y} + cy$$





## Discrete time implementation

Software implementation of the controller



Discrete-time sliding mode control

$$\overline{\mathbf{x}}[k+1] = \mathbf{A}_{d}\overline{\mathbf{x}}[k] + \mathbf{B}_{d}\mathbf{\sigma}[k]$$

$$\mathbf{\sigma}[k+1] = \mathbf{\Phi}_{d}(\overline{\mathbf{x}}[k], \mathbf{\sigma}[k], k) + \mathbf{\Gamma}_{d}\mathbf{u}[k]$$

$$\mathbf{u}[k] = -\alpha \operatorname{sgn}(\mathbf{\sigma}[k])$$

Discrete-time sliding mode control can appear also in systems with continuous right-hand-side (e.g., deadbeat control)

Discrete time sliding mode control is sensible only in the presence of uncertainties or disturbances

## Discrete time implementation

What is a discrete time sliding mode?

$$\sigma[k+1] = 0$$
  $k = 0,1,2,...$ 

$$\sigma[k+1] - \sigma[k] = 0$$
  $k = 0,1,2,...$ 

The second is not convincing and does not imply the first

The usual implementation of the control law has two parts

- \* the nominal part
- \* the discontinuous part to cope with uncertainties

# Discrete time implementation

Usual accuracy of discrete time sliding modes is O (T<sub>c</sub>)

Learning and adaptive methods



Multirate sampling allows for output-feedback implementation of the controller

Effective approach is constituted by continuous time design and subsequent discretization analysis



Can chaotic behaviour appear ?!

The sliding mode concept can be extended to more complete integral manifolds

$$\sigma = \dot{\sigma} = \ddot{\sigma} = \cdots = \sigma^{(r-1)} = 0$$

- r constraints are introduced in the system dynamics
- The control acts on the r-th derivative of the sliding variable  $\sigma$
- Accuracy is improved

$$\left|\sigma^{(i)}\right| = H_i \tau^{r-i}, \quad i = 0, 1, 2, \dots, r-1$$

 $H_{
m i}$  does not depend on au

- $\triangleright$  The internal dynamics has a reduced dimension n-r
- The robustness properties are preserved
- The approximability property is preserved
- Allows for chattering attenuation
- ➤ No general Lyapunov-like methods for the stability analysis are available (only very recent results for r=2 were presented)
- Phase plane of homogeneity approaches to state the closed-loop stability
- $\triangleright$  Derivatives of the sliding variable  $\sigma$  are needed

Higher Order Sliding Modes can be used to regularise nonlinear variable structure systems with sliding modes

$$\dot{\mathbf{x}} = \mathbf{f}(\mathbf{x}, u(t), t)$$

$$\mathbf{x} \in R^{n} \quad \sigma \in R \quad u \in R$$

$$\sigma = \mathbf{h}(\mathbf{x}, t)$$

$$\dot{\sigma} = \frac{\partial h(\mathbf{x}, t)}{\partial t} + \frac{\partial \mathbf{h}(\mathbf{x}, t)}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}, u(t), t) = \phi(\mathbf{x}, u(t), t)$$

$$\ddot{\sigma} = \frac{\partial \phi(\mathbf{x}, u(t), t)}{\partial t} + \frac{\partial \phi(\mathbf{x}, u(t), t)}{\partial \mathbf{x}} \mathbf{f}(\mathbf{x}, u(t), t) + \frac{\partial \phi(\mathbf{x}, u(t), t)}{\partial u} \dot{u} = \phi(\mathbf{x}, u(t), t) + \gamma(\mathbf{x}, u(t), t) v$$

If a discontinuous v constrains the system onto the  $2^{nd}$  order manifold

$$\sigma = \dot{\sigma} = 0$$

Filippov's solution and the "Equivalent Control" coincide, and the control u is continuous (**Chattering attenuation**)

The control algorithms for Higher-Order Sliding Modes require derivatives of  $\sigma$ 

Relative degree	Feedback signals	VSS controller
1	sign(s)	Classical 1-SMC, Super-Twisting
2	$s,\dot{s}$	Dynamic / Terminal 2-SMC
	$s, \{sign(\dot{s})\}$	Sub-optimal 2-SMC
	$sign(s)$ , $sign(\dot{s})$	Twisting 2-SMC
3	$sign(s)$ , $sign(\dot{s})$ , $sign(\ddot{s})$	Hybrid 3-VSC
Any r	$s, \dot{s},, s^{(r-2)}, sign(s^{(r-1)})$	Universal HOSM

#### Dynamical sliding modes

$$s = x_n + \sum_{i=1}^{n-1} c_i x_i$$

$$\sigma = s^{(r-1)} + \sum_{i=1}^{r-2} \gamma_i s^{(i-1)}$$

Starting from a classical sliding variable s, a new switching variable  $\sigma$  is defined

Sliding mode is achieved asymptotically

Terminal sliding modes

$$s = x_n + \sum_{1}^{n-1} c_i x_i$$

$$1 < \frac{p}{q} < 2, \ p, q \text{ odd}$$

$$\sigma = s + \gamma \dot{s}^{\frac{p}{q}}$$

A nonlinear switching surface is defined Sliding mode is achieved in finite time

#### Super-Twisting controller

$$\dot{\sigma} = \varphi(\bullet) + \gamma(\bullet)u$$

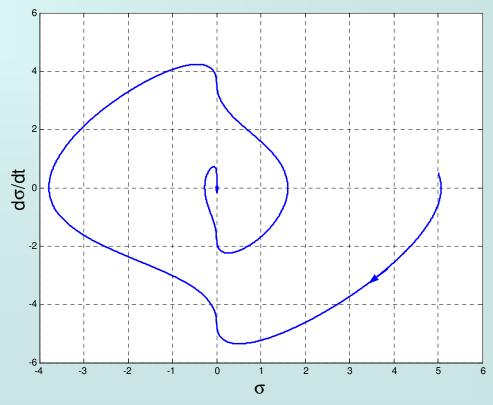
$$u = -\alpha \sqrt{|\sigma|} + z \qquad \lambda > \frac{\Phi}{\Gamma_m}$$

$$\dot{z} = -\lambda \operatorname{sgn}(\sigma) \qquad \alpha^2 \ge \frac{4\Phi}{\Gamma_m^2} \frac{\Gamma_M(\lambda + \Phi)}{\Gamma_m(\lambda - \Phi)}$$

Φ: upper bound of the drift term

 $\Gamma_{\rm m}$ ,  $\Gamma_{\rm M}$ : positive lower and upper bounds of the gain term

The control is continuous



Suboptimal-controller

 $\ddot{\sigma} = \varphi(\bullet) + \gamma(\bullet)u$ 

$$u = -\alpha(t)U \operatorname{sgn}(\sigma - \beta \sigma_{M})$$

$$\alpha(t) = \begin{cases} 1 & (\sigma - \beta \sigma_{M})\sigma_{M} \ge 0 \\ \overline{\alpha} & (\sigma - \beta \sigma_{M})\sigma_{M} < 0 \end{cases}$$

$$\beta \in (0;1)$$

$$u = -\alpha(t)U\operatorname{sgn}(\sigma - \beta\sigma_{M})$$

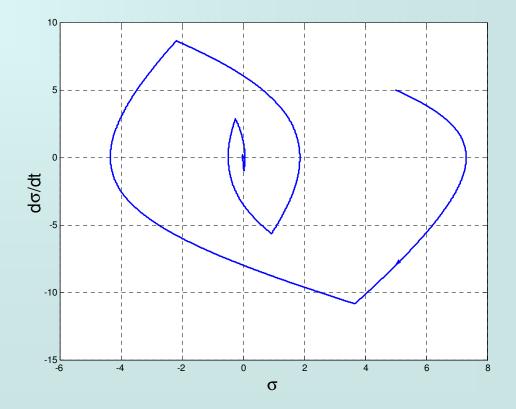
$$\alpha(t) = \begin{cases} 1 & (\sigma - \beta\sigma_{M})\sigma_{M} \ge 0 \\ \overline{\alpha} & (\sigma - \beta\sigma_{M})\sigma_{M} < 0 \end{cases}$$

$$\overline{\alpha} \in [1; +\infty) \cap \left(\frac{2\Phi + \Gamma_{M}U(1 - \beta)}{\Gamma_{m}U(1 + \beta)}; +\infty\right)$$

Φ: upper bound of the drift term

 $\Gamma_{\rm m}$ ,  $\Gamma_{\rm M}$ : positive lower and upper bounds of the gain term

The control is discontinuous



#### Suboptimal-controller

$$u = -\alpha(t)U\operatorname{sgn}(\sigma - \beta\sigma_{M})$$

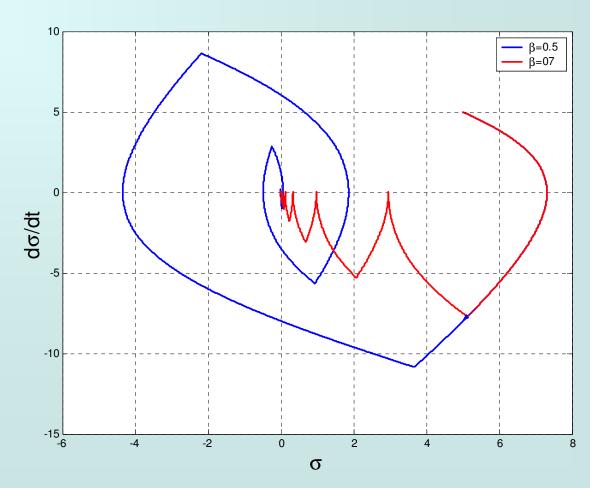
$$\alpha(t) = \begin{cases} 1 & (\sigma - \beta\sigma_{M})\sigma_{M} \ge 0 \\ \overline{\alpha} & (\sigma - \beta\sigma_{M})\sigma_{M} < 0 \end{cases}$$

$$\beta \in (0;1)$$

β: anticipation factor

 $\alpha$ : modulation factor

U: control gain



#### Universal quasi-continuous HOSM-controller

$$u = -\alpha \Psi_{n-1,n}(e_y, \dot{e}_y, ..., e_y^{(n-1)}).$$

$$e_y = \sigma$$

$$\begin{array}{lcl} \varphi_{0,n} & = & e_y, \ N_{0,n} = |e_y| \\ \Psi_{0,n} & = & \varphi_{0,n}/N_{0,n} = \mathrm{sign} \ e_y, \\ \varphi_{i,n} & = & e_y^{(i)} + \beta_i N_{i-1,n}^{(n-i)/(n-i+1)} \Psi_{i-1,n}, \\ N_{i,n} & = & |e_y^{(i)}| + \beta_i N_{i-1,n}^{(n-i)(n-i+1)}, \\ \Psi_{i,n} & = & \varphi_{i,n}/N_{i,n} \end{array}$$

Homogeneous nested implementations of discontinuous controllers

#### Final remarks

- ✓ Sliding Modes are a usual behaviour in switching systems
- ✓ Sliding Modes are a useful tool for controlling uncertain dynamical systems
- ✓ Switching control with Sliding Modes is a simple way of applying the Internal Model Principle
- ✓ By resorting to the Equivalent Control definition it is possible to retrieve some information about an uncertain system by low-pass filters
- ✓ Ideal Sliding Modes are not implemented in practice because they require infinite frequency switching, and only an approximate sliding can be achieved
- ✓ Higher Order Sliding Modes generalise the concept of sliding modes to integral manifolds
- ✓ Real Higher Order Sliding Modes improve the accuracy and attenuate the chattering effect