

Supercritical CO2 axial turbine design for CSP applications.

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Background

Concentrated-solar power (CSP) systems with a maximum efficiency of 42% can be achieved using Rankine cycles.

Supercritical carbon dioxide (sCO2) power promising candidates cycles applications for CSP plants with an efficiency > 50%.

sCO2 blends are required for plants located in dry areas

Objectives

To establish a design methodology for supercritical axial turbine design for sCO2 working fluid blends.

To evaluate the validity of the existing loss models for the sCO2 turbomachines using computational fluid dynamics (CFD) simulations.

To examine the off-design performance of sCO2 turbomachines under CSP system operating conditions.

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Methodology

Mean-line Design

- 1D mean-line design integrated with suitable loss models.
- sCO2 properties: The Peng-Robinson Equation of State.

CFD Simulations

Single flow passage.

Steady mixingplane conditions for the interface between the rotor and the stator.

Verification Lack of sCO₂ turbines' prototypes.

Challenging design validation.

100 MW sCO₂ axial turbine design has been used to verify the design model.

Design Parameters:

Flow coefficient φ ,

Loading coefficient ψ ,

Degree of reaction Λ ,

Isentropic efficiency η,

Nozzle loss coefficient,

rotational speed N.

Design A - Zoomed in $b_2 = 0.73 \text{ mm}$ $b_3 = 1.29 \text{ mm}$ Design B - Zoomed in r = 27.00 mm $b_2 = 2.03 \text{ mm}$ $b_3 = 2.23 \text{ mm}$ r = 20.00 mm100-kW sCO2 axial turbine design at (A) specified design point and (B) optimized design for feasible micro turbine dimensions.

Conclusions

100 kW small-scale turbine design has been developed for the design of small scale axial sCO2 turbines.

Next steps:

- Extend the mean-line model to 100 MW design.
- Conduct 3D CFD simulations to further verify the model.
- Conduct off-design performance analysis for CSP applications

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Mean-line Design

and Design parameters selections Compute blade geometry, velocity triangles, blade angles. Compute centrifugal tensile and gas bending stresses and blade speed NO YES Check YES design constraints

Input boundary constraints

Mean-line

Design

Model

validation

CFD-3D

simulations

Compute losses and turbine efficiency

> η -Assumed η |< 0.05

End

Design Constraints: Mean blade speed. Allowable stress limit.

Loss Models:

Soderberg and Craig and cox