

# Effect of Governor Deadbands on Valve Travel using Long-Term Dynamic Simulation

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Abstract—Abstract goes here.

Index Terms—Governor deadbands, long-term dynamic simulation, time-sequenced power flow, valve travel

## I. Introduction

Deadbands are commonly implemented in turbine speed governors to prevent a machine from responding to small frequency deviations that are ever present in electrical systems. Industry usage of deadbands is widely known, however, it is often overlooked in power system simulation. [1] details how incorporating governor deadbands into transient simulation models can lead to results that better match measured power system events.

mish mash intro thoughts:

- deadband simulation considered in transient stability time frame, not long-term
- [2] reports frequency profile improvement caused by no step deadband. Also uses MW-min calculation to state smaller no step deadbands reduce movement, calls for immediate dev of gov guidelines - led to standard TRE BAL-001 for ERCOT - in full effect april 2015. Most Recent Report [3] has odd ERCOT frequency probability - more U shaped, less normal.
- transient simulation not meant for long term simulation
- long term dynamic simulation software focusing on governor dynamics was proposed in [4] and others, but has not been standardized nor fully developed.

This paper explores the impact of governor deadbands using a novel long term simulation environment based on GE Energy's Positive Sequence Load Flow (PSLF). The paper proposes a cumulative valve travel metric for comparing the impact of various deadband scenarios. Section II describes the long term simulation environment used to perform the studies. Section III discusses governor and deadband modeling within the simulation environment. In Section IV, the long term simulation is validated against mature transient stability simulations. Finally, Section V presents some initial results demonstrating the viability of the proposed valve travel metric and Section VI offers conclusions.

## II. Long-Term Dynamic Simulation Technique

Time-sequenced power flow (TSPF) is a method for long-term dynamic (LTD) simulation proven to generate

useful results [5]. The basic idea behind TSPF is to solve a power flow, perform system dynamics of interest, 're-seed' the power flow with new values, and repeat. A python based simulation software, Power System Long-Term Dynamic Simulator (PSLTDSim), has been developed to perform LTD simulations using TSPF. PSLTDSim has the ability to calculate system frequency, perform governor dynamics, model automatic generation control (AGC), and insert step, ramp, and noise type perturbances into a power system.

## A. Simulation Assumptions and Simplifications

Due to the relatively large time steps of 1 second involved with TSPF, numerous assumptions were made. Ideal exciters are assumed as modern exciters are typically fast enough to maintain reference voltage under stable conditions. Intermachine oscillations are ignored since the time resolution used is not fine enough to capture these phenomena.

Simplifications of transient stability models are used in PSLTDSim. The only parameters required to model a generator are MW cap, MVA base, and machine inertia. Additionally, a deadband modified tgov1 governor model, Fig. 1, was created to model system governors.

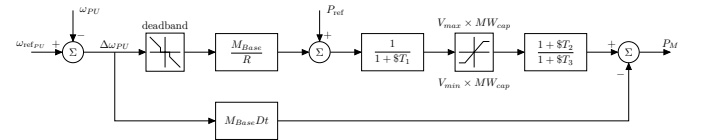


Fig. 1. Block diagram of modified tgov1 model.

## B. Modeling the System-Wide Frequency Response

Instead of frequency being calculated for every bus, a combined swing equation is used to model only one system frequency. As shown in (1), accelerating power from the entire system, as well as total system inertia  $H_{PU,sys}$ , is used to calculate  $\dot{\omega}_{sys}$ .

$$\dot{\omega}_{sys} = \frac{1}{2H_{PU,sys}} \left( \frac{P_{accPU,sys}}{\omega_{sys}} - D_{sys}\Delta\omega_{sys} \right) \quad (1)$$

### C. Distribution of Accelerating Power

In a system with  $N$  generators, total system accelerating power is calculated by

$$P_{acc,sys} = \sum_{i=1}^N P_{m,i} - \sum_{i=1}^N P_{e,i} \quad (2)$$

where  $P_{m,i}$  is mechanical power and  $P_{e,i}$  is electrical power of generator  $i$ .

System accelerating power is distributed to all generators in the system according to machine inertia as

$$P_{e,i} = P_{e,i} - P_{acc,sys} \left( \frac{H_i}{H_{sys}} \right) \quad (3)$$

where  $H$  has units of  $MW \cdot s$ .

The new value for  $P_{e,i}$  is used in the next power flow solution for each generator. If the difference between expected and resulting power supplied by the slack generator is larger than a set slack tolerance, the difference is redistributed according to (3) until the resulting difference is below the slack tolerance, or a maximum number of iterations take place [6].

### III. Implementation of Deadbands

FERC minimum requirements for droop and deadband are 5% and 36 mHz respectively[7]. However, practical implementation is left to generator operators.

#### A. Types of Deadbands

Fig. 2 presents implementations of deadbands that follow FERC requirements. If a governor has no deadband, a change in output power is requested for any frequency deviation. A step deadband ignores any frequency smaller than the setpoint  $\pm db_1$  and then steps to meet the set droop curve. A no step deadband pushes the original droop curve away from the nominal frequency allowing for the droop curve to cross zero at  $\pm db_1$ . A non-linear deadband linearly increases from  $\pm\alpha$  to  $\pm\beta$ , after which it follows the original droop curve.

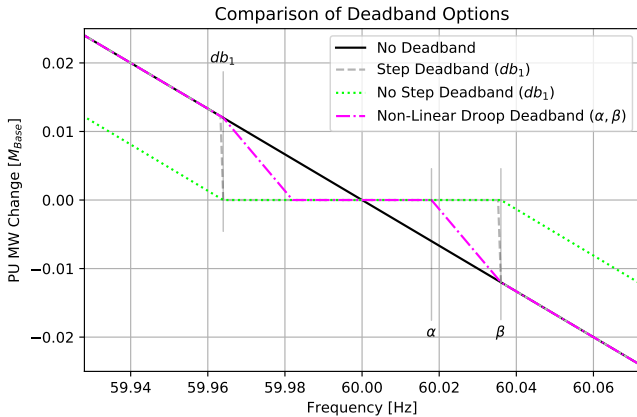


Fig. 2. Different types of deadbands.

### IV. Simulation Validation

To validate the chosen simulation approach, identical system perturbances were performed in the GE Energy Positive Sequence Load Flow (PSLF) Dynamic Subsystem (PSDS) and PSLTDSim. Frequency is compared to validate the single system frequency assumption calculated by PSLTDSim, and governed generator mechanical power is compared to validate governor action.

To simplify the comparison of frequency data from PSDS to LTD, a single weighted frequency based on generator inertia was calculated using (4) and (5).

$$f_w = \sum_{i=1}^N f_i \frac{H_{PU,i} M_{base,i}}{H_{sys}} \quad (4)$$

$$\text{where } H_{sys} = \sum_{i=1}^N H_{PU,i} M_{base,i} \quad (5)$$

Due to the different time steps, when calculating the difference between PSDS and LTD, multiple PSDS values have the same held LTD value subtracted from them. For instance, any  $PSDS(t = 3.x)$  would have the  $LTD(t = 3)$  value subtracted from it for comparison.

#### A. The MiniWECC System

The power system used for validation and valve travel experiments, the miniWECC shown in Fig. 3, is a 120 bus 34 generator system created in PSLF. All governors in the miniWECC are modeled with the `tgov1` which enabled easier validation. Further details about the creation and use of the miniWECC may be found in [8]–[10].

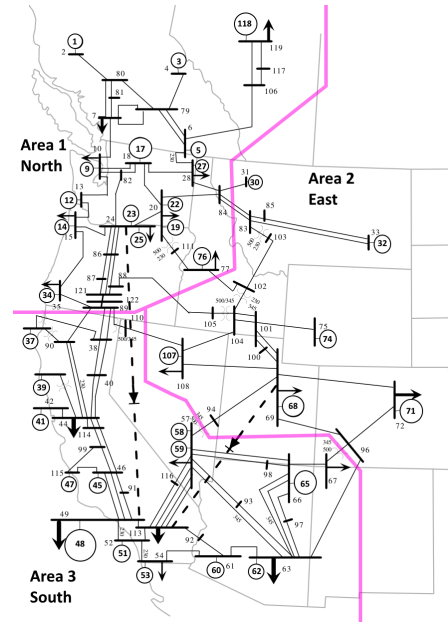


Fig. 3. MiniWECC System adapted from [10].

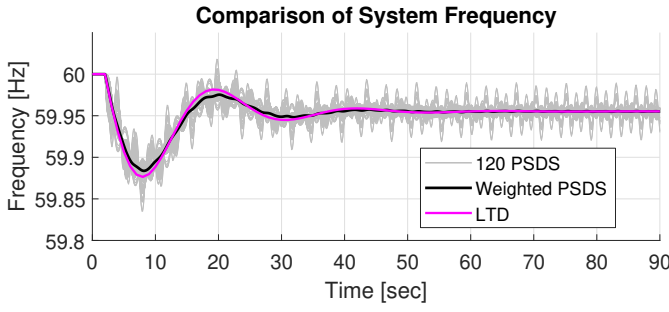


Fig. 4. Comparison of frequency during load step.

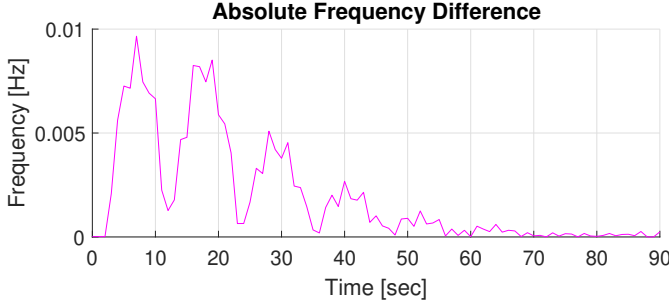


Fig. 5. Absolute difference of weighted frequency during load step.

1) Load Step Results: A 400 MW load step was simulated to occur at 2 seconds. As shown in Fig. 4, all individual PSDS frequencies begin to oscillate after the perturbation while the weighted PSDS frequency appears to follow the general center of oscillation. The LTD system frequency is less oscillatory than the weighted frequency with only minor differences between the two. Fig. 9 quantifies these differences.

When comparing mechanical power output in Fig. 6, large MW differences can be seen, however, the percent difference data in Fig. 7 shows results less than 5% max difference, and an average percent difference of less than  $\approx 0.5\%$ .

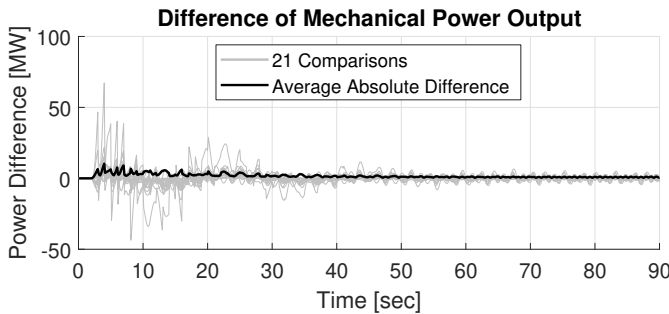


Fig. 6. Difference of mechanical power output during load step.

2) Load Ramp Results: Simulation results from a 40 second 400 MW load ramp, in Fig. 8-10 show frequency of LTD being within 1.2 mHz of PSDS and mechanical power differences of less than  $\pm 10$  MW or 1% difference

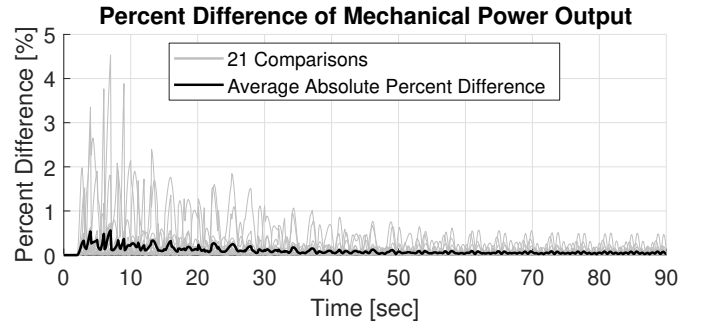


Fig. 7. Percent difference of mechanical power output during load step.

max (0.2% average).

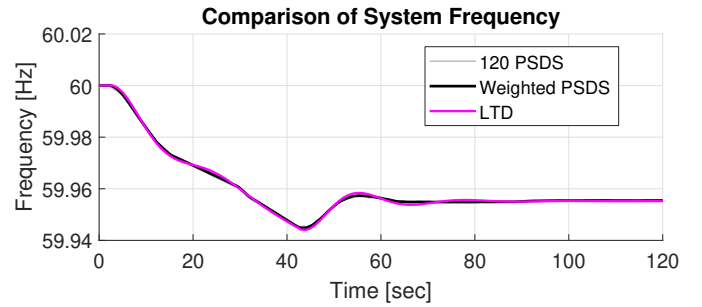


Fig. 8. Comparison of frequency during load ramp.

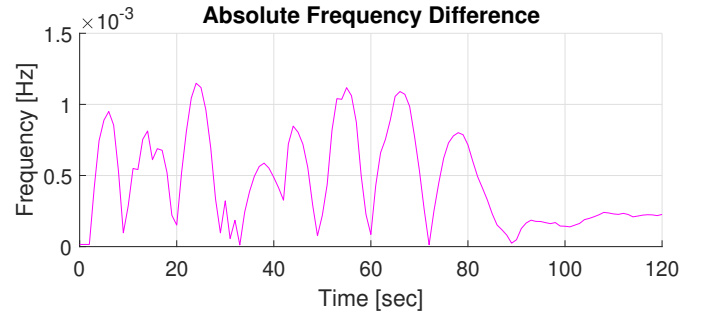


Fig. 9. Absolute difference of weighted frequency during load ramp.

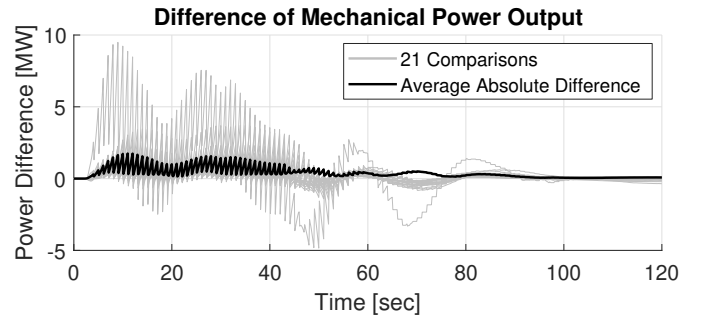


Fig. 10. Difference of mechanical power output during load ramp.

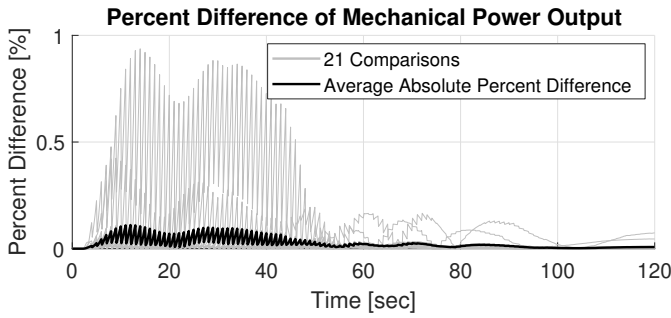


Fig. 11. Percent difference of mechanical power output during load ramp.

## B. Validation Summary

The step tests show the software is not meant to simulate large transient events with great accuracy. However, small perturbances are modeled with less deviation from transient simulation methods.

### V. Initial Results

Explanation of simulations to do...

30 minutes of noise with various deadbands and resulting valve travel. Cases of interest: No deadband, step deadband, no-step deadband, nl-droop deadband.

Universal acceptance sim: At least 1 area with a step deadband, others with linear or no deadband. Should show unequal distribution of valve travel.

#### A. System Modifications

The miniWECC system was modified to include three areas for AGC simulation. PSLTDSim was used to set all area governor deadbands to 5%. Governors were removed from some units so that only  $\approx 20\%$  of generation capacity in each area has governor control. A generic AGC routine that filters area control error (ACE) through a proportional-integral (PI) controller was added to each area. AGC messages are sent select generation units with governors representing  $\approx 9\%$  of total generation capacity every 15 seconds.

#### B. Simulation Results

To verify desirable AGC control operation, a simple 1500 MW generation loss event was simulated...

... Noise test frequency plots and valve travel movement, table of results

### VI. Conclusion

Basic theme: Maybe having deadbands is a bad move - an extra work caused by procrastination type of situation. But everyone probably needs to be on board, otherwise some may ride free while others unfairly pick up the slack.

#### Acknowledgment

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## References

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- [11] P. Kundur, *Power System Stability and Control*. McGraw-Hill, 1994.

## VII. Non-Text Format Templates

This section meant to provide template figures, tables, equations, and references that can be copied and pasted to other parts of the document in a simple manner.

Figures will behave as Fig. 12. Note that the placement may seem random, but is chosen by  $\text{\LaTeX}$  automatically.

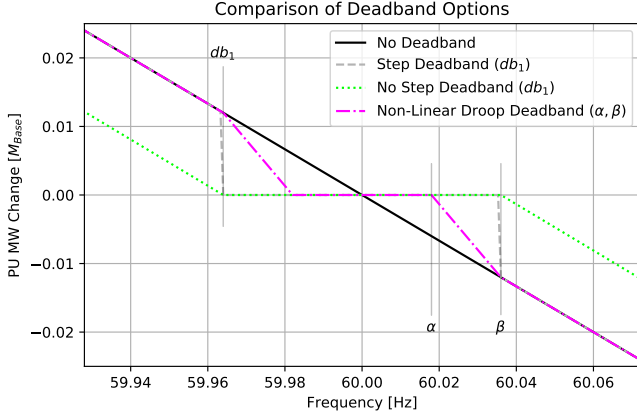


Fig. 12. Testing of figure format.

The default IEEE table example leaves something to be desired that is fulfilled by using the booktabs package. This is shown in Table I.

TABLE I  
Generic governor model parameters.

Parameter	Steam	Hydro	Gas
Ts	0.04	0.40	0.50
Tc	0.20	45.00	10.00
T3	0.00	5.00	4.00
T4	1.50	-1.00	0.00
T5	5.00	0.50	1.00

Equations are entered as one may normally do in a  $\text{\LaTeX}$  situation and referenced as (6) and (7).

$$f_{ss} = f_{ref} + \Delta f = f_{ref} + \frac{\Delta P}{S_{Base}\beta} \quad (6)$$

$$\beta = \sum_{i=1}^N \frac{1}{R_i \frac{S_{Base}}{M_{Base,i}}} \quad (7)$$

References are only included if cited. For instance [11] or [5] are randomly cited. Note that the sorting order is set to none, which lists references in order cited.