

# AAE 590: Lie Group Methods for Control and Estimation

## Group Affine Systems and Geometric Control

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## **Group Affine Systems and Geometric Control**

- Group affine systems and autonomous error dynamics
- Geometric control and PID on  $\text{SE}(2)$
- Fixed-wing aircraft trajectory tracking (2D)
- Log-linear dynamic inversion

*From theory to application!*

# Motivation: Linear Systems Have Autonomous Error Dynamics

**Linear system:**  $\dot{x} = Ax + Bu, \quad \dot{\bar{x}} = A\bar{x} + B\bar{u}$

**Error:**  $e = x - \bar{x} \quad \Rightarrow \quad \dot{e} = Ae + Bu_e \quad \checkmark \text{ Autonomous in } e!$

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**Nonlinear counterexample:** **Unicycle** with  $\dot{x} = u \cos \theta, \dot{y} = u \sin \theta, \dot{\theta} = \omega$

Error  $e = (x - \bar{x}, y - \bar{y}, \theta - \bar{\theta})$ :

$$\dot{e}_x = u \cos \theta - \bar{u} \cos \bar{\theta} = u \cos(\bar{\theta} + e_\theta) - \bar{u} \cos \bar{\theta}$$

This depends on  $\bar{\theta}$ , not just  $e!$  **X Not autonomous!**

## The Punchline

On  $\text{SE}(2)$ :  $\dot{X} = X\xi^\wedge$  with  $\xi = (u, 0, \omega)^T$  is **left-invariant**  $\Rightarrow$  **group affine**!

Same physical system, but error  $\eta = \bar{X}^{-1}X$  has **autonomous** dynamics!

# Group Affine Systems: Definition [1]

## Definition

A vector field  $f_u : G \rightarrow TG$  is **group affine** if for all  $X, Y \in G$ :

$$f_u(XY) = f_u(X)Y + Xf_u(Y) - Xf_u(I)Y$$

**Running example — Unicycle on  $SE(2)$ :**

$$\dot{X} = X\xi^\wedge, \quad \xi = (u, 0, \omega)^T \quad (\text{left-invariant} \Rightarrow \text{group affine!})$$

## Where Does This Come From?

This definition isn't arbitrary—it's **derived** from requiring autonomous error dynamics. The next slides show Barrau & Bonnabel's elegant derivation.

## Deriving the Group Affine Property (1/3) [1]

**Goal:** Find what property  $f_u$  must satisfy for error  $\eta = \hat{X}^{-1}X$  to have **autonomous** dynamics  $\dot{\eta} = g(\eta)$ .

**Setup:** True and estimated states evolve by  $\dot{X} = f_u(X)$  and  $\dot{\hat{X}} = f_u(\hat{X})$ .

**Step 1: Differentiate the error using product rule**

$$\begin{aligned}\dot{\eta} &= \frac{d}{dt}(\hat{X}^{-1}X) = \frac{d}{dt}(\hat{X}^{-1}) \cdot X + \hat{X}^{-1} \cdot \dot{X} \\ &= -\hat{X}^{-1}\dot{\hat{X}}\hat{X}^{-1}X + \hat{X}^{-1}\dot{X}\end{aligned}$$

**Step 2: Substitute the dynamics**

$$\dot{\eta} = -\hat{X}^{-1}f_u(\hat{X})\hat{X}^{-1}X + \hat{X}^{-1}f_u(X)$$

Since  $\eta = \hat{X}^{-1}X$ , we have  $X = \hat{X}\eta$ , so:

$$\boxed{\dot{\eta} = -\hat{X}^{-1}f_u(\hat{X})\eta + \hat{X}^{-1}f_u(\hat{X}\eta)}$$

## Deriving the Group Affine Property (2/3) [1]

**Key question:** When does  $\dot{\eta} = -\hat{X}^{-1}f_u(\hat{X})\eta + \hat{X}^{-1}f_u(\hat{X}\eta)$  depend **only on**  $\eta$ ?

**Step 3: Assume autonomous form exists**

Suppose  $\dot{\eta} = g(\eta)$  for some function  $g$ . We can find  $g$  by setting  $\hat{X} = I$ :

$$\begin{aligned} g(\eta) &= -I^{-1}f_u(I)\eta + I^{-1}f_u(I \cdot \eta) \\ &= -f_u(I)\eta + f_u(\eta) \end{aligned}$$

The Autonomous Error Dynamics

$$g(\eta) = f_u(\eta) - f_u(I)\eta$$

This is the form error dynamics **must** take if they are autonomous!

## Deriving the Group Affine Property (3/3) [1]

### Step 4: Derive the constraint on $f_u$

For  $\dot{\eta} = g(\eta) = f_u(\eta) - f_u(I)\eta$  to hold for **all**  $\hat{X}$ , substitute back:

$$f_u(\eta) - f_u(I)\eta = -\hat{X}^{-1}f_u(\hat{X})\eta + \hat{X}^{-1}f_u(\hat{X}\eta)$$

Multiply both sides by  $\hat{X}$  and rearrange:

$$f_u(\hat{X}\eta) = f_u(\hat{X})\eta + \hat{X}f_u(\eta) - \hat{X}f_u(I)\eta$$

**Renaming**  $\hat{X} \rightarrow X$  and  $\eta \rightarrow Y$ :

$$f_u(XY) = f_u(X)Y + Xf_u(Y) - Xf_u(I)Y$$

### The Punchline

The group affine property is **exactly** what's needed for autonomous error dynamics—not an arbitrary definition, but a **derived necessity!**

## Worked Example: Unicycle is Group Affine

Show the unicycle  $\dot{X} = X\xi^\wedge$  with  $\xi = (u, 0, \omega)^T$  satisfies:  $f_u(XY) = f_u(X)Y + Xf_u(Y) - Xf_u(I)Y$ .

## Solution: Unicycle is Group Affine

For  $f_u(X) = X\xi^\wedge$  (left-invariant):

**LHS:**  $f_u(XY) = (XY)\xi^\wedge$

**RHS:**  $f_u(X)Y + Xf_u(Y) - Xf_u(I)Y = X\xi^\wedge Y + XY\xi^\wedge - X\xi^\wedge Y = XY\xi^\wedge \checkmark$

**Key:**  $f_u(I) = \xi^\wedge$ , so the correction term  $-Xf_u(I)Y$  cancels  $f_u(X)Y$ .

### The Unicycle Takeaway

The “nonlinear” unicycle is actually **linear in the Lie algebra** and **group affine**!

# Mixed Invariant Vector Fields [1]

## Definition

A **mixed invariant** vector field:  $f_u(X) = X\xi_L(u) + \xi_R(u)X$  where  $\xi_L(u), \xi_R(u) \in \mathfrak{g}$  depend only on inputs  $u$ , **not on state  $X$** .

## Special Cases and Error Frames

- **Left-invariant:**  $f_u(X) = X\xi_L \Rightarrow$  error  $\eta = \hat{X}^{-1}X$  in **body frame**
- **Right-invariant:**  $f_u(X) = \xi_R X \Rightarrow$  error  $\eta = X\hat{X}^{-1}$  in **world frame**

## Choose Intelligently!

Pick the error definition that matches your sensor frame:

- IMU (body-frame measurements)  $\rightarrow$  left-invariant error
- GPS/landmarks (world-frame)  $\rightarrow$  right-invariant error

## Mixed Invariant $\Rightarrow$ Group Affine [1]

**Theorem:** If  $f_u(X) = X\xi_L + \xi_R X$ , then  $f_u$  is group affine.

**Proof:** Verify  $f_u(XY) = f_u(X)Y + Xf_u(Y) - Xf_u(I)Y$ :

$$\text{LHS: } f_u(XY) = XY\xi_L + \xi_R XY$$

$$\begin{aligned}\text{RHS: } &= (X\xi_L + \xi_R X)Y + X(Y\xi_L + \xi_R Y) - X(\xi_L + \xi_R)Y \\ &= X\xi_L Y + \xi_R XY + XY\xi_L + X\xi_R Y - X\xi_L Y - X\xi_R Y \\ &= XY\xi_L + \xi_R XY \quad \checkmark\end{aligned}$$

### Open Question

Converse (group affine  $\Rightarrow$  mixed invariant) is conjectured but not proven.

# Autonomous Error Dynamics [1]

**Theorem:** If  $f_u$  is group affine, error  $\eta = \hat{X}^{-1}X$  has **autonomous** dynamics.

**Setup:**  $\dot{X} = f_u(X)$ ,  $\dot{\hat{X}} = f_u(\hat{X})$ ,  $X = \hat{X}\eta$ . **Error derivative:**

$$\dot{\eta} = -\hat{X}^{-1}f_u(\hat{X})\eta + \hat{X}^{-1}f_u(X)$$

**Apply group affine** to  $f_u(X) = f_u(\hat{X}\eta)$ :

$$f_u(\hat{X}\eta) = f_u(\hat{X})\eta + \hat{X}f_u(\eta) - \hat{X}f_u(I)\eta$$

**Substitute and simplify:**

$$\dot{\eta} = -\hat{X}^{-1}f_u(\hat{X})\eta + \hat{X}^{-1}[f_u(\hat{X})\eta + \hat{X}f_u(\eta) - \hat{X}f_u(I)\eta] = \boxed{f_u(\eta) - f_u(I)\eta}$$

# The Magic: Trajectory Independence [1]

**Result:** For group affine systems:  $\dot{\eta} = f_u(\eta) - f_u(I)\eta$

**Unicycle example:**  $f_u(X) = X\xi^\wedge$ , so  $f_u(I) = \xi^\wedge$  and:

$$\dot{\eta} = \eta\xi^\wedge - \xi^\wedge\eta = [\eta, \xi^\wedge] \quad (\text{commutator!})$$

## Why This Matters

Error dynamics depend **only on** the error  $\eta$  and input  $\xi$ .

**Not on** the true pose  $X$  or estimate  $\hat{X}$  — unlike Cartesian coordinates!

## Consequences for estimation:

- Linearized error dynamics are **state-independent**
- EKF Jacobians don't depend on state estimate
- Filter covariance is **consistent** (doesn't grow incorrectly)

## This is the IEKF Foundation

The Invariant EKF exploits this property for superior convergence!

# Geodesics and Group Affine Systems

## Definition

A **geodesic** on Lie group  $G$ :  $\gamma(t) = \gamma(0) \cdot \text{Exp}((t\xi)^{\wedge})$  for fixed  $\xi \in \mathfrak{g}$ . The “straightest” curves on  $G$ .

**Key:** Geodesics = constant Lie algebra velocities.

**For left-invariant**  $\dot{X} = X\xi^{\wedge}$ : if  $\xi$  constant, trajectory is geodesic.

## Caveat

Autonomous error requires both trajectories follow **same**  $f_u$ . For time-varying  $\xi(t)$ , not geodesics, but group affine still ensures state-independent linearization.

# Explicit Error Dynamics for Mixed Invariant [1]

For **mixed invariant**  $f_u(X) = X\xi_L + \xi_R X$ , substitute into  $\dot{\eta} = f_u(\eta) - f_u(I)\eta$ :

$$f_u(\eta) = \eta\xi_L + \xi_R\eta, \quad f_u(I) = \xi_L + \xi_R, \quad f_u(I)\eta = \xi_L\eta + \xi_R\eta$$

Therefore:

$$\dot{\eta} = \eta\xi_L + \xi_R\eta - \xi_L\eta - \xi_R\eta = \boxed{\eta\xi_L - \xi_L\eta = [\eta, \xi_L]}$$

## Key Result

For **left-invariant** systems ( $\xi_R = 0$ ): error dynamics are the **commutator**  $[\eta, \xi_L]$ —linear in the Lie algebra near identity!

# Why “Linear in the Lie Algebra”?

Group error dynamics  $\dot{\eta} = \eta\xi^\wedge - \bar{\xi}^\wedge\eta$  are nonlinear on  $G$ , but...

**Parameterize near identity:**  $\eta = \text{Exp}(\epsilon^\wedge)$ ,  $\epsilon \in \mathfrak{g}$

**Lie algebra error dynamics:**

$$\dot{\epsilon} = A(\bar{\xi})\epsilon + (\xi - \bar{\xi}) + O(\|\epsilon\|^2)$$

## The Key Property

$A(\bar{\xi})$  depends **only on inputs**, not on state! This gives:

- **Linear** error propagation (for small errors)
- **State-independent** Jacobians
- **Consistent** filter covariance

# SE(2) Kinematics is Group Affine

Fixed-wing aircraft (unicycle) kinematics:

$$\dot{X} = X \underbrace{\begin{bmatrix} 0 & -\omega & v \\ \omega & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}}_{\xi^\wedge}$$

This is **purely left-invariant**:  $\dot{X} = X\xi^\wedge$  with  $\xi_R = 0$

Why left-invariant?

- $\xi = (v, 0, \omega)^T$  is the **body-frame** velocity
- Aircraft moves forward relative to its own heading
- Control inputs  $(v, \omega)$  are body-referenced

## Group Affine Structure

Since  $\xi$  depends only on inputs  $(v, \omega)$ , not on state  $X$ , this system is group affine!

# Autonomous Error Dynamics on SE(2)

**Consider:** Actual trajectory  $X(t)$  and reference  $\bar{X}(t)$ , both satisfying:

$$\dot{X} = X\xi^\wedge, \quad \dot{\bar{X}} = \bar{X}\bar{\xi}^\wedge$$

**Define error:**  $\eta = \bar{X}^{-1}X$  (left-invariant error)

**Error dynamics:**

$$\begin{aligned}\dot{\eta} &= \frac{d}{dt}(\bar{X}^{-1}X) = -\bar{X}^{-1}\dot{\bar{X}}\bar{X}^{-1}X + \bar{X}^{-1}\dot{X} \\ &= -\bar{X}^{-1}\bar{X}\bar{\xi}^\wedge\bar{X}^{-1}X + \bar{X}^{-1}X\xi^\wedge \\ &= -\bar{\xi}^\wedge\eta + \eta\xi^\wedge\end{aligned}$$

**Result:**

$$\boxed{\dot{\eta} = \eta\xi^\wedge - \bar{\xi}^\wedge\eta}$$

# Why Autonomous Error Dynamics Matter

$$\dot{\eta} = \eta \xi^\wedge - \bar{\xi}^\wedge \eta$$

**Key observation:** Error dynamics depend on:

- The error  $\eta$  itself
- The control input  $\xi$
- The reference input  $\bar{\xi}$

**NOT on the absolute states  $X$  or  $\bar{X}$ !**

## Consequences

- **Control design:** Same controller works everywhere on the manifold
- **Stability analysis:** Error equilibrium at  $\eta = I$  (identity)
- **Linearization:** Jacobians are state-independent
- **Filtering:** Consistent covariance propagation (Week 9)

## Worked Example: Error Dynamics on SE(2)

Robot at  $(4, 3), 0$ ; reference at  $(2, 1), 0$ . Compute  $\eta = \bar{X}^{-1}X$  and  $\xi_e = \text{Log}(\eta)^\vee$ .

## Solution: Error Dynamics on SE(2)

$$\bar{X} = \begin{bmatrix} 1 & 0 & 2 \\ 0 & 1 & 1 \\ 0 & 0 & 1 \end{bmatrix}, \quad \bar{X}^{-1} = \begin{bmatrix} 1 & 0 & -2 \\ 0 & 1 & -1 \\ 0 & 0 & 1 \end{bmatrix}$$
$$\eta = \bar{X}^{-1} X = \begin{bmatrix} 1 & 0 & -2 \\ 0 & 1 & -1 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} 1 & 0 & 4 \\ 0 & 1 & 3 \\ 0 & 0 & 1 \end{bmatrix} = \begin{bmatrix} 1 & 0 & 2 \\ 0 & 1 & 2 \\ 0 & 0 & 1 \end{bmatrix} \quad (\text{pure translation error})$$

$\omega = 0, \quad \mathbf{t}_{\text{error}} = (2, 2)^T$ . Since  $\omega = 0$ :  $V(0)^{-1} = I$

$\xi_e = \text{Log}(\eta)^\vee = (2, 2, 0)^T \quad (2 \text{ units right, 2 units up in reference frame})$

# Linearized Error Dynamics

Near the identity,  $\eta \approx I + \epsilon^\wedge$  where  $\epsilon \in \mathbb{R}^3$  is small.

**Linearizing**  $\dot{\eta} = \eta \xi^\wedge - \bar{\xi}^\wedge \eta$ :

$$\dot{\epsilon}^\wedge \approx (I + \epsilon^\wedge) \xi^\wedge - \bar{\xi}^\wedge (I + \epsilon^\wedge)$$

To first order:

$$\dot{\epsilon}^\wedge = \xi^\wedge - \bar{\xi}^\wedge + \epsilon^\wedge \xi^\wedge - \bar{\xi}^\wedge \epsilon^\wedge$$

**In vector form:**  $\dot{\epsilon} = A\epsilon + (\xi - \bar{\xi})$

where  $A$  depends on  $\bar{\xi}$  but **NOT on  $X$  or  $\bar{X}$ !**

## State-Independent Jacobians

This is why PID control works uniformly across all of SE(2)—the linearization looks the same everywhere.

# Group Affine: The Big Picture

**This week:** Group affine structure on  $\text{SE}(2)$

- Definition and key property
- Autonomous error dynamics
- Foundation for control (next section)

**Weeks 8–9:** Group affine systems for estimation

- Extended pose group  $\text{SE}_2(3)$  with mixed invariance
- Invariant EKF exploits autonomous error dynamics
- State-independent Jacobians  $\Rightarrow$  consistent filter

## Key Takeaway

Group affine structure is the theoretical foundation for both geometric control **and** invariant filtering.

# The Control Problem [2]

## Goal

Design a controller to track a reference trajectory  $\bar{X}(t) \in \text{SE}(2)$

## Given:

- Current pose:  $X \in \text{SE}(2)$
- Reference pose:  $\bar{X}(t) \in \text{SE}(2)$
- Control inputs: forward velocity  $v$ , yaw rate  $\omega$

**Design:** Feedback law  $(v, \omega) = f(X, \bar{X})$

## Challenge

How do we define “error” when states live on a curved manifold?

# Why Euclidean Error Fails

**Naive approach:**  $e = (x - \bar{x}, y - \bar{y}, \theta - \bar{\theta})$

**Problems:**

- ① **Angle wrapping:**  $\theta = 179^\circ$  and  $\bar{\theta} = -179^\circ$  gives  $e_\theta = 358^\circ$ !
- ② **Frame inconsistency:** Position error in world frame, but control in body frame
- ③ **Ignores geometry:** Doesn't respect the group structure

## Key Insight

The error should be computed **using the group operation**, not subtraction.

# Group Error: Left vs. Right

**Two natural choices:**

**Left-invariant error:**

$$\eta_L = \bar{X}^{-1} X$$

“Error from reference’s perspective” (body frame of reference)

**Right-invariant error:**

$$\eta_R = X \bar{X}^{-1}$$

“Error from current pose’s perspective” (body frame of vehicle)

Both are valid!

Choice depends on where you want to express the error.

# Understanding Left-Invariant Error

**Left-invariant error:**  $\eta_L = \bar{X}^{-1}X$

**Interpretation:**

- $\eta_L$  describes how to get from  $\bar{X}$  to  $X$
- Expressed in the **reference frame**
- At zero error:  $\eta_L = I$  (identity)

If  $\eta_L = \begin{bmatrix} R_e & \mathbf{t}_e \\ 0 & 1 \end{bmatrix}$ :

- $R_e$ : orientation error (how much to rotate)
- $\mathbf{t}_e$ : position error in reference frame

# Understanding Right-Invariant Error

**Right-invariant error:**  $\eta_R = X\bar{X}^{-1}$

**Interpretation:**

- $\eta_R$  describes how to get from  $\bar{X}$  to  $X$
- Expressed in the **current body frame**
- At zero error:  $\eta_R = I$  (identity)

If  $\eta_R = \begin{bmatrix} R_e & \mathbf{t}_e \\ 0 & 1 \end{bmatrix}$ :

- $R_e$ : same orientation error
- $\mathbf{t}_e$ : position error in current body frame

# Which Error to Use?

**For vehicle control: Right-invariant is often preferred**

**Why?**

- Controls  $(v, \omega)$  are in body frame
- Body-frame error directly maps to control action
- “I need to go forward and turn left” makes sense in body frame

**Alternative view:** Use left-invariant with reference-frame controls, then transform.

**Important:** This is a Choice!

Both conventions are valid and used in the literature:

- **Left-invariant:**  $\eta_L = \bar{X}^{-1}X$  (error in reference frame)
- **Right-invariant:**  $\eta_R = X\bar{X}^{-1}$  (error in body frame)

We use **left-invariant** in this course. Be consistent and explicit!

# Error in the Lie Algebra

**Group error:**  $\eta = \bar{X}^{-1} X \in \text{SE}(2)$

**Algebra error:**  $\xi_e = \text{Log}(\eta)^\vee \in \mathbb{R}^3$

$$\xi_e = \begin{bmatrix} x_e \\ y_e \\ \theta_e \end{bmatrix}$$

## Properties:

- $\xi_e = 0$  when  $X = \bar{X}$  (perfect tracking)
- $\xi_e$  is a vector — we can do linear algebra!
- Automatically handles angle wrapping
- Captures position error in reference frame

# Computing the Error Twist

Given:

$$X = \begin{bmatrix} R & \mathbf{t} \\ 0 & 1 \end{bmatrix}, \quad \bar{X} = \begin{bmatrix} \bar{R} & \bar{\mathbf{t}} \\ 0 & 1 \end{bmatrix}$$

**Step 1: Compute group error**

$$\eta = \bar{X}^{-1}X = \begin{bmatrix} \bar{R}^T R & \bar{R}^T(\mathbf{t} - \bar{\mathbf{t}}) \\ 0 & 1 \end{bmatrix}$$

**Step 2: Apply logarithm**

$$\xi_e = \text{Log}(\eta)^\vee = \begin{bmatrix} x_e \\ y_e \\ \theta_e \end{bmatrix}$$

where  $\theta_e = \text{atan2}(\eta_{21}, \eta_{11})$  and  $(x_e, y_e)^T = V(\theta_e)^{-1}\eta_{1:2,3}$

# Proportional Control

**Simplest controller:**  $u = -K_p \xi_e$

$$\begin{bmatrix} v_x \\ v_y \\ \omega \end{bmatrix} = -K_p \begin{bmatrix} x_e \\ y_e \\ \theta_e \end{bmatrix}$$

**For a unicycle (no lateral motion):**

$$\omega = -k_\theta \theta_e$$

$$v = -k_x x_e$$

**Issue:** What about  $y_e$ ?

Unicycle Constraint

Unicycles can't move sideways!  $v_y = 0$  always. We need a smarter approach.

# Dealing with Non-Holonomic Constraints

**Unicycle:**  $v_y = 0$  (can't move laterally)

**Solution:** Use rotation to eliminate lateral error

**Intuition:**

- If target is to the left, turn left first
- Drive forward to reduce  $x_e$
- Turning handles  $y_e$  indirectly

**Modified control law:**

$$\omega = -k_\theta \theta_e - k_y y_e$$

$$v = -k_x x_e$$

The  $k_y y_e$  term turns toward the target!

## PID Control in Lie Algebra [2, 3]

**Full PID law:**

$$u = -K_p \xi_e - K_i \int_0^t \xi_e(\tau) d\tau - K_d \dot{\xi}_e$$

**Gain matrices:** (with  $\xi_e = (v_{ex}, v_{ey}, \omega_e)^T$ ,  $u = (v, \omega)^T$ )

$$K_p = \begin{bmatrix} k_{p,x} & 0 & 0 \\ 0 & k_{p,\theta y} & k_{p,\theta} \end{bmatrix}, \quad K_i, K_d \text{ similar}$$

**For unicycle:** Output is  $(v, \omega)^T \in \mathbb{R}^2$ , input is  $\xi_e \in \mathbb{R}^3$

**Key insight:** The  $K_{p,\theta y}$  term couples yaw control to lateral error!

# Computing $\dot{\xi}_e$

For the derivative term, we need  $\dot{\xi}_e$

**Option 1:** Numerical differentiation

$$\dot{\xi}_e \approx \frac{\xi_e(t) - \xi_e(t - \Delta t)}{\Delta t}$$

**Option 2:** Analytical (more complex)

$$\dot{\xi}_e = J_I^{-1}(\xi_e) (\xi - \text{Ad}_{\eta^{-1}} \bar{\xi})$$

where  $J_I$  is the left Jacobian of the exponential map.

**In practice:** Numerical differentiation with filtering works well.

# Stability Analysis

**Linearization at  $\eta = I$ :**

Near identity,  $\text{Log}(\eta) \approx \eta - I$  and dynamics become linear:

$$\dot{\xi}_e \approx A\xi_e + Bu$$

**For proportional control:**

$$\dot{\xi}_e \approx (A - BK_p)\xi_e$$

**Stability:** Eigenvalues of  $(A - BK_p)$  in left half-plane

## Local Stability

With proper gain selection, the system is locally asymptotically stable around  $\eta = I$ .

## Worked Example: Linearized Stability

*For P-control on SE(2) with  $K_p = \text{diag}(1, 1, 2)$ , find eigenvalues of closed-loop system.*

## Solution: Linearized Stability

For fully actuated system:  $\dot{\xi}_e = -K_p \xi_e$  (ignoring ref feedforward)

Closed-loop:  $A_{cl} = -K_p = \begin{bmatrix} -1 & 0 & 0 \\ 0 & -1 & 0 \\ 0 & 0 & -2 \end{bmatrix}$

Eigenvalues:  $\lambda_1 = -1, \lambda_2 = -1, \lambda_3 = -2$

All eigenvalues  $< 0 \Rightarrow \text{asymptotically stable}$

Heading converges 2× faster than position (by design with  $K_p$ ).

# Why Lie Algebra PID Works

## Key properties:

- ① **Well-defined error:**  $\xi_e = 0 \Leftrightarrow X = \bar{X}$
- ② **Smooth near identity:** Log is smooth for small errors
- ③ **Coordinate-free:** No gimbal lock or singularities
- ④ **Natural gains:** Physical interpretation of  $K_p$  entries

## Limitations

- Large errors: Log has limited domain
- Global stability not guaranteed (only local)
- Non-holonomic constraints need special handling

# Feedforward + Feedback

**Better approach: Feedforward + feedback**

**Reference trajectory provides:**

- $\bar{X}(t)$ : desired pose
- $\bar{\xi}(t)$ : desired twist (feedforward)

**Control law:**

$$u = u_{ff} + u_{fb} = \bar{\xi} - K_p \xi_e - K_d \dot{\xi}_e$$

**Interpretation:**

- $\bar{\xi}$ : “what we should be doing”
- $-K_p \xi_e$ : “correction for position error”
- $-K_d \dot{\xi}_e$ : “damping”

# Body Frame vs. Reference Frame

**Our error  $\xi_e$  is in reference frame**

**Controls are in body frame**

**Transform:** Use the adjoint!

$$\xi_{body} = \text{Ad}_{\eta}^{-1} \xi_{ref\_frame}$$

**Or equivalently:** The error twist in body frame is:

$$\xi_e^{body} = \text{Log}(X^{-1}\bar{X})^\vee = -\text{Log}(\eta^{-1})^\vee$$

## Practical Note

For small errors, the difference between frames is small. Many implementations ignore this.

# Summary: Geometric Control Recipe

## Step 1: Define error

$$\eta = \bar{X}^{-1} X$$

## Step 2: Map to algebra

$$\xi_e = \text{Log}(\eta)^\vee$$

## Step 3: Apply PID

$$u = \bar{\xi} - K_p \xi_e - K_i \int \xi_e - K_d \dot{\xi}_e$$

## Step 4: Handle constraints

- Project to feasible controls
- Couple  $\omega$  with lateral error for non-holonomic vehicles

# Log-Linear Dynamic Inversion [4]

*Lin, Goppert, Hwang (2024) — IEEE Trans. Automatic Control*

**Key insight:** Group affine structure enables exact linearization via Jacobians!

**Error dynamics on group:**  $\dot{\eta} = \eta\xi^\wedge - \bar{\xi}^\wedge\eta$

**Error dynamics in algebra:** Let  $z = \text{Log}(\eta)^\vee$ , then:

$$\dot{z} = J_I^{-1}(z) \cdot \underbrace{(\xi - \text{Ad}_\eta^{-1}\bar{\xi})}_{\text{"effective input"}}$$

**The magic:** Choose control to cancel the Jacobian:

$$\xi = \bar{\xi} + J_I(z) \cdot v \quad \Rightarrow \quad \dot{z} = v$$

## LTI Closed-Loop!

Set  $v = -Kz$  and get  $\dot{z} = -Kz$  — **exactly** linear, not just linearized!

This works because group affine  $\Rightarrow$  Jacobians depend only on error  $z$ , not on state  $X$ .

## Worked Example: Log-Linear Dynamic Inversion

For group affine system  $\dot{\eta} = \eta\xi^\wedge - \bar{\xi}^\wedge\eta$ , design control so error dynamics become LTI:  $\dot{z} = -Kz$ .

## Solution: Log-Linear Dynamic Inversion

**Step 1:** Error in Lie algebra:  $z = \text{Log}(\eta)^\vee \in \mathbb{R}^3$

**Step 2:** Differentiate using chain rule with Jacobian:

$$\dot{z} = J_I^{-1}(z) \cdot (\text{body velocity of } \eta) = J_I^{-1}(z) \cdot (\xi - \text{Ad}_\eta^{-1}\bar{\xi})$$

**Step 3:** Near identity ( $z$  small):  $\text{Ad}_\eta^{-1} \approx I - \text{ad}_z$ , so:

$$\dot{z} \approx J_I^{-1}(z) \cdot (u + \text{ad}_z\bar{\xi}) \quad \text{where } u = \xi - \bar{\xi}$$

**Step 4:** Log-linear dynamic inversion — choose  $u$  to cancel nonlinearity:

$$u = J_I(z) \cdot (-Kz) - \text{ad}_z\bar{\xi}$$

**Result:**  $\dot{z} = -Kz$  (LTI closed-loop!) With  $K = 2I$ :  $z(t) = z_0 e^{-2t}$

# Summary

## This lecture we learned:

- Group affine systems and autonomous error dynamics
- Left/right invariant vector fields: body vs world frame
- Geometric control: PID in the Lie algebra
- Log-linear dynamic inversion for exact linearization

## Key takeaway:

Lie Algebra = Natural Control Space

Working in the Lie algebra gives us a vector space where PID makes sense, while respecting the geometry of the configuration space.

## Next Week: SO(3)

### Week 4: The Special Orthogonal Group SO(3)

- 3D rotations — non-abelian!
- Rodrigues formula for Exp
- Axis-angle and quaternion representations
- Connection to aerospace attitude

**Preview:** SO(3) is like SO(2) but with non-trivial Ad and Lie bracket. Everything we learned for SE(2) applies, but in 3D!

## References I

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- [4] Li-Yu Lin, James Goppert, and Inseok Hwang. "Log-Linear Dynamic Inversion Control With Provable Safety Guarantees in Lie Groups". In: *IEEE Transactions on Automatic Control* 69.8 (2024), pp. 5591–5597. DOI: 10.1109/TAC.2024.3369549.