

$PV^\gamma = C$ is valid for an ideal gas undergoing reversible adiabatic process.

Conventional Problem

Q \Rightarrow A fluid is contained in a cylinder by a spring loaded frictionless piston so that the pressure in fluid is a linear function of volume i.e. $P = a + bV$, where a & b are constants. internal energy of the fluid is given by $U = 34 + 3.15PV$ where U is in KJ, P in KPa & V in m^3 . if the fluid changes from initial state of $P_1 = 170$ KPa, $V_1 = 0.03 m^3$ to a final state of $P_2 = 400$ KPa, $V_2 = 0.06 m^3$ find the magnitude & direction of heat & work t/y.

Sol \Rightarrow $W = \frac{1}{2}(170 + 400)(0.06 - 0.03)$

$$W = 8.55 \text{ KJ}$$

$$\pm Q = dU + \pm W$$

$$U_2 = 34 + 3.15 P_2 V_2$$

$$U_1 = 34 + 3.15 P_1 V_1$$

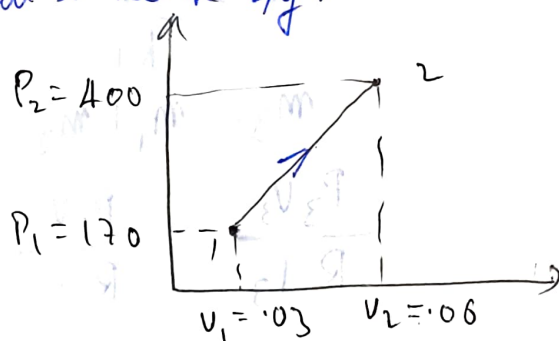
$$U_2 - U_1 = 3.15 (P_2 V_2 - P_1 V_1)$$

$$dU = 3.15 [400 \times 0.06 - 170 \times 0.03]$$

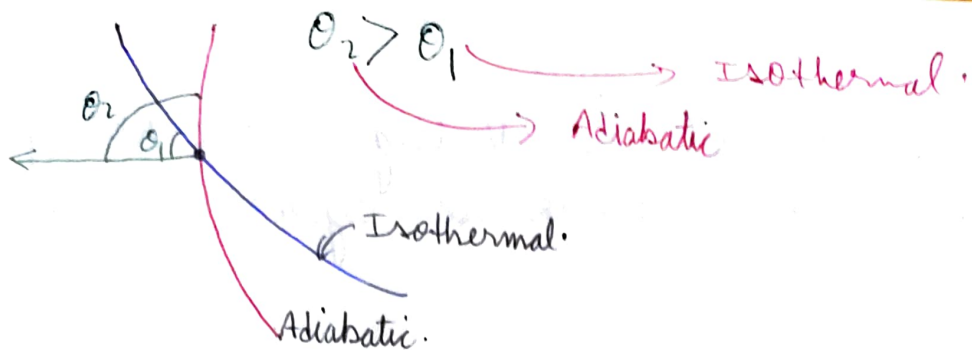
$$dU = 59.535 \text{ KJ}$$

$$\pm Q = 59.535 + 8.55$$

$$\pm Q = 68.085 \text{ KJ}$$



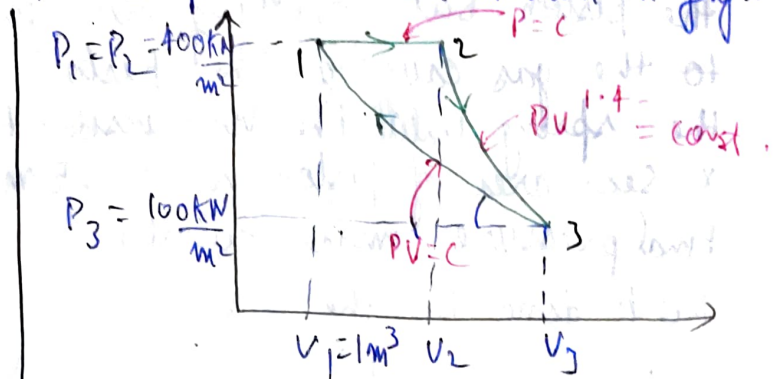
Q \Rightarrow An insulated rigid pressure vessel is divided into two portions by a partition. The first part of the vessel is occupied by an ideal gas at a pressure P_1 , Vol V_1 & Temperature T_1 . The other part is occupied by the same ideal gas but at pressure P_2 , Vol V_2 , Temp. T_2 . Suddenly the partition is removed & two portions mix with each other. Show that P_3 & T_3 are given by



Conventional Problems

A system undergoes three processes as shown in figure

- Find (i) V_2 .
(ii) Net work \pm / y .



Solⁿ \Rightarrow Process (3-1) $T = C$

$$P_3 V_3 = P_1 V_1 \Rightarrow \boxed{V_3 = 4 \text{ m}^3}$$

Process (2-3) $P V^{1.4} = C$

$$P_2 V_2^{1.4} = P_3 V_3^{1.4} \Rightarrow \boxed{V_2 = 1.486 \text{ m}^3}$$

WD for 1-2

$$W_{1-2} = P(V_2 - V_1) = 400 \times (1.486 - 1) = 194.4 \text{ KJ}$$

WD for 2-3

$$W_{2-3} = \frac{P_2 V_2 - P_3 V_3}{\gamma - 1} = \frac{400 \times 1.486 - 100 \times 4}{1.4 - 1} = 486 \text{ KJ}$$

WD for 3-1

$$W_{3-1} = P_3 V_3 \ln\left(\frac{V_1}{V_3}\right) = -554.5 \text{ KJ}$$

$$\text{Net work} = W_{1-2} + W_{2-3} + W_{3-1} = 125.86 \text{ KJ} = 125.86 \text{ KJ}$$

Concluding Remarks \Rightarrow Net work in a cycle = area of closed region.

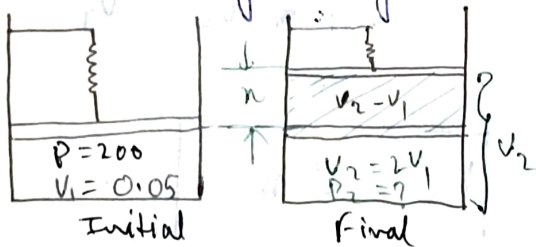
All clockwise cycle on P-V diagram are work producing cycles & all anticlockwise cycles are work absorbing cycles.

Q2 \Rightarrow A piston cylinder device contains 0.05 m^3 of a gas initially at 200 kPa at this state a linear spring which has a spring const. of 150 kN/m^2 is just touch the piston but exerting no force on it. Heat is t/ied to the gas causing the piston to rise & to compress the spring until the vol. inside the cylinder doubles if the x-sec. area of piston is 0.25 m^2 find-

(i) Final pressure inside cylinder

(ii) Work done by the gas.

Solⁿ \Rightarrow



$$V_2 - V_1 = Ax$$

$$2V_1 - V_1 = Ax$$

$$V_1 = Ax$$

$$x = \frac{V_1}{A} \Rightarrow \frac{0.05}{0.25}$$

$$x = 0.2 \text{ m}$$

From initial FBD

$$P_{\text{atm}} + \frac{W}{A} = 200$$

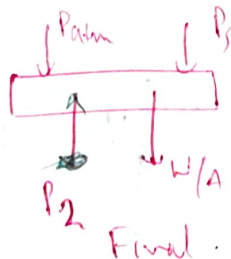
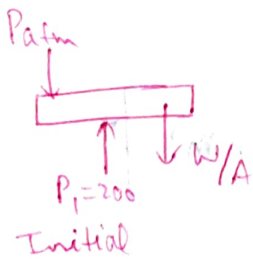
From Final FBD

$$P_{\text{atm}} + P_s + \frac{W}{A} = P_2 \Rightarrow P_{\text{atm}} + P_s + \frac{W}{A} = P_2$$

$$P_s + 200 = P_2$$

$$P_s = \frac{F_s}{A} = \frac{k \cdot x}{A} = 120 \text{ kPa}$$

$$\Rightarrow P_2 = 320 \text{ kPa}$$



$$P_2 = 200 + P_s$$

$$P_s = \frac{F_s}{A} = \frac{Kx}{A}$$

$$V_2 - V_1 = Ax$$

$$\Delta V = Ax$$

$$x = \frac{\Delta V}{A}$$

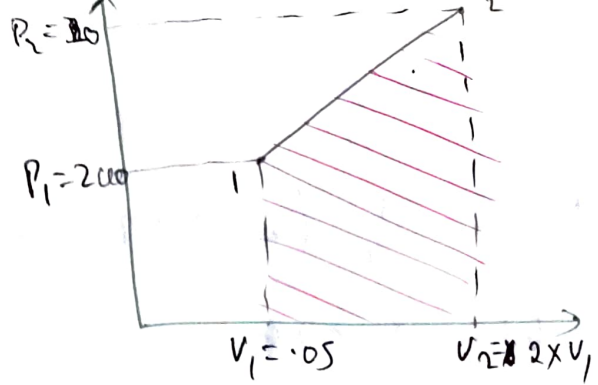
$$P_s = \frac{K}{A} \frac{\Delta V}{A}$$

$$P_2 = 200 + \frac{K \Delta V}{A^2}$$

$K, A = \text{const} \Rightarrow \text{Linear.}$

$$W = \frac{1}{2} (200 + 320) (0.1 - 0.05)$$

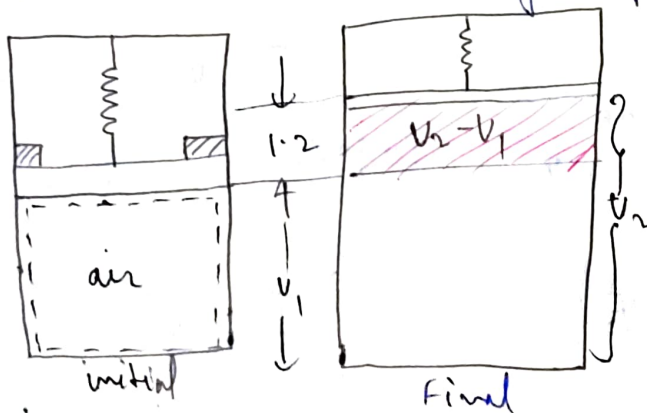
$$W = 13 \text{ KJ}$$



Q=

A closed cylinder of 0.25 m dia fitted with a light frictionless piston. The piston is retained in posⁿ by stops in cylinder wall. The vol. on one side of piston contains air at a pressure of 750 kN/m². The vol. on the other side of the piston is evacuated. A spring is mounted in this evacuated space to give a force of 120 N on piston in this posⁿ. The stops are removed & the piston travels along the cylinder until it comes to rest after a stroke of 1.2 meters. The piston is then held in this posⁿ. The spring force increases linearly to final value of 5 kN. Calculate the work done by compressed air on piston.

Solⁿ ⇒



$$P_1 = \frac{F_1}{A} = \frac{120 \text{ N}}{\frac{\pi}{4} (0.25)^2}$$

$$= 2444.6 \text{ N/m}^2$$

$$= 2.444 \text{ kN/m}^2$$

$$P_2 = \frac{F_2}{A} = \frac{5 \text{ kN}}{\frac{\pi}{4} (0.25)^2}$$

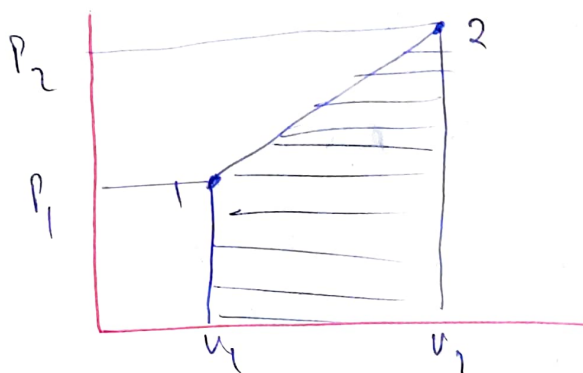
$$P_2 = 101.85 \text{ kN/m}^2$$

$$V_2 - V_1 = A \Delta x$$

$$V_2 - V_1 = \frac{\pi}{4} \times (25)^2 \times 0.2 \Rightarrow V_2 - V_1 = 0.0589 \text{ m}^3$$

$$W = \frac{1}{2} (2.444 + 101.85) \times 0.0589$$

$$W = 3.02 \text{ kJ}$$



Q \Rightarrow An ideal gas is heated at const. vol. until its temp. is three times the original temperature it is then expanded isothermally till it reaches original pressure the gas is then cooled at const pressure till it is restored to the original state. Determine the net work done by the gas / Kg. the initial temp is 350 K. Express your ans in terms of gas constant R.

$$\text{Ans} \Rightarrow 453.5R$$

$$\text{Sol}^n \Rightarrow T_1 = 350 \text{ K}$$

$$T_2 = T_3 = 3T_1 = 1050 \text{ K}$$

$$W_{D \text{ for } 31} = P_1(V_3 - V_2)$$

$$PV = nRT$$

$$PV = RT$$

$$P_1 V_1 = RT_1 = 350R$$

$$P_2 V_2 = RT_2 = 1050R$$

$$P_3 V_3 = RT_3 = 1050R$$

$$\Rightarrow V_2 = \frac{1050R}{P_2}$$

$$\Rightarrow V_3 = \frac{1050R}{P_3}$$

$$P_2 V_2 = nRT_2$$

$$\Rightarrow \frac{P_3}{P_2} = \frac{V_2}{V_3}$$

$$W_{D \text{ for } 23} = P_2 V_2 \ln\left(\frac{V_2}{V_3}\right) = P_2 V_2 \ln\left(\frac{P_3}{P_2}\right) = 1153.54R$$

$$W_{D \text{ for } 13} = P_3(V_3 - V_2) = P_3 \left(\frac{1050R}{P_3} - \frac{1050R}{P_2} \right) = 1050R \left(1 - \frac{P_3}{P_2} \right) = 700R$$

$$W_{\text{net}} = 1153.54R + 700R = 453.54R$$

