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Analysis of a Parallel and Counterflow Concentric Heat Exchanger

MECH 257 Presentation

Presented by Andrew Lemus



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Goal

Model and simulate a parallel and counterflow concentric heat exchanger.

Additionally:

1. Learn how to analyze a thermal+fluid model in COMSOL
2. Consider mesh sizes along with computation time
3. Compare the results of the two heat exchangers

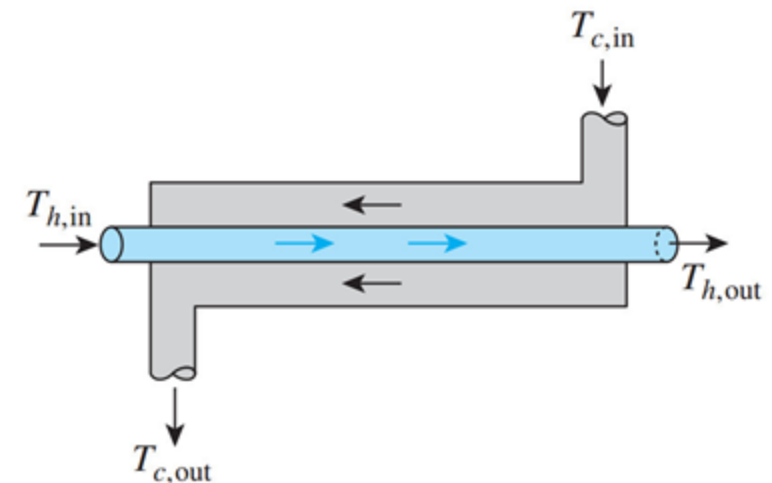


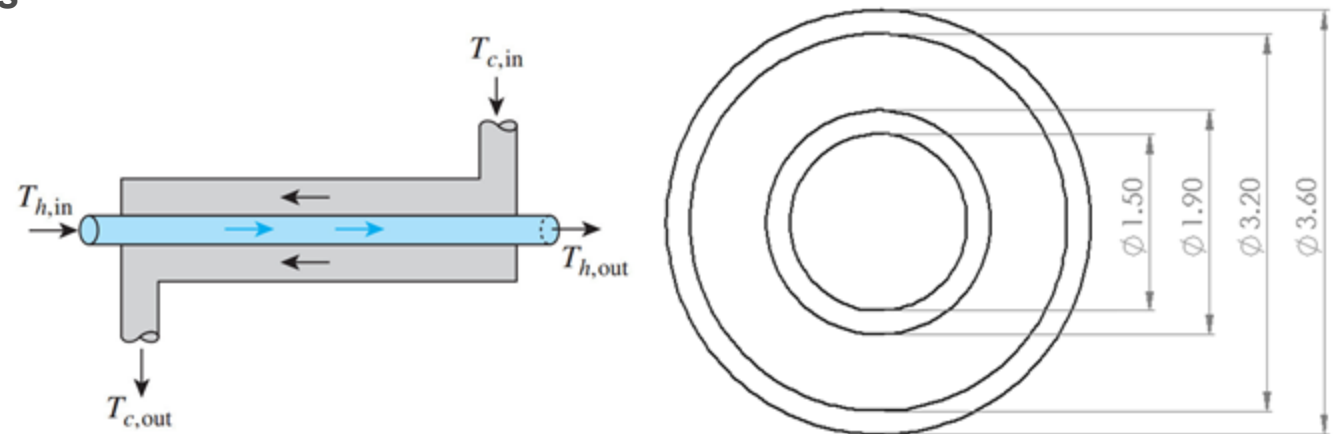
Image from <https://www.thomasnet.com/articles/process-equipment/double-pipe-heat-exchangers/>



Parameters

- Length, $L = 10$ meters
- Hot and cold flow, $U = 1$ m/s
- Hot water input, $T_{h,in} = 100^\circ\text{C}$
- Cold water input, $T_{c,in} = 20^\circ\text{C}$
- The shell thickness does not matter for this simulation
 - Convection can be applied to outside if desired

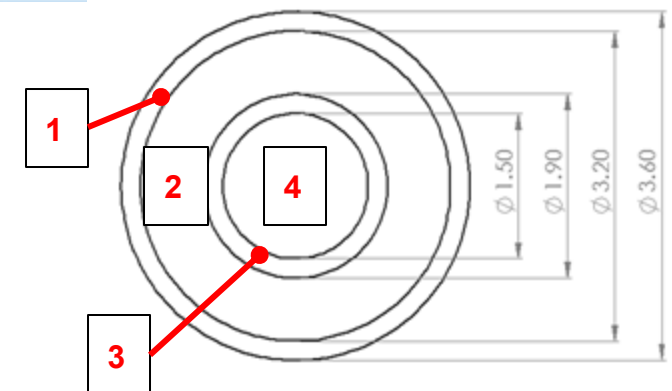
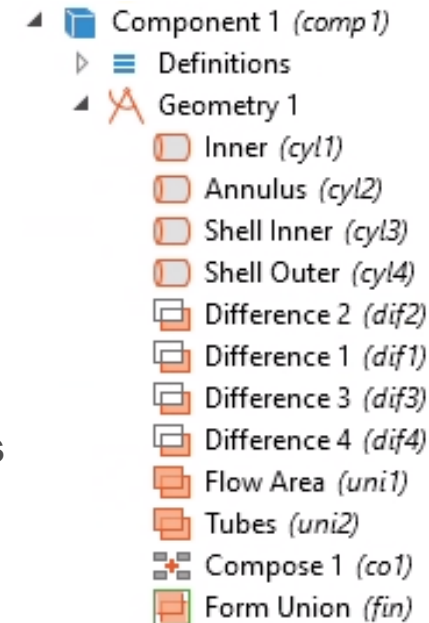
Name	Expression	Value	Description
Di	1.5[cm]	0.015 m	
Da	1.9[cm]	0.019 m	
Dshell_i	3.2[cm]	0.032 m	
Dshell_o	3.6[cm]	0.036 m	
L	10[m]	10 m	
Ucold_in	1[m/s]	1 m/s	
Uhot_in	1[m/s]	1 m/s	
Thot_in	100[degC]	373.15 K	
Tcold_in	20[degC]	293.15 K	





Geometries

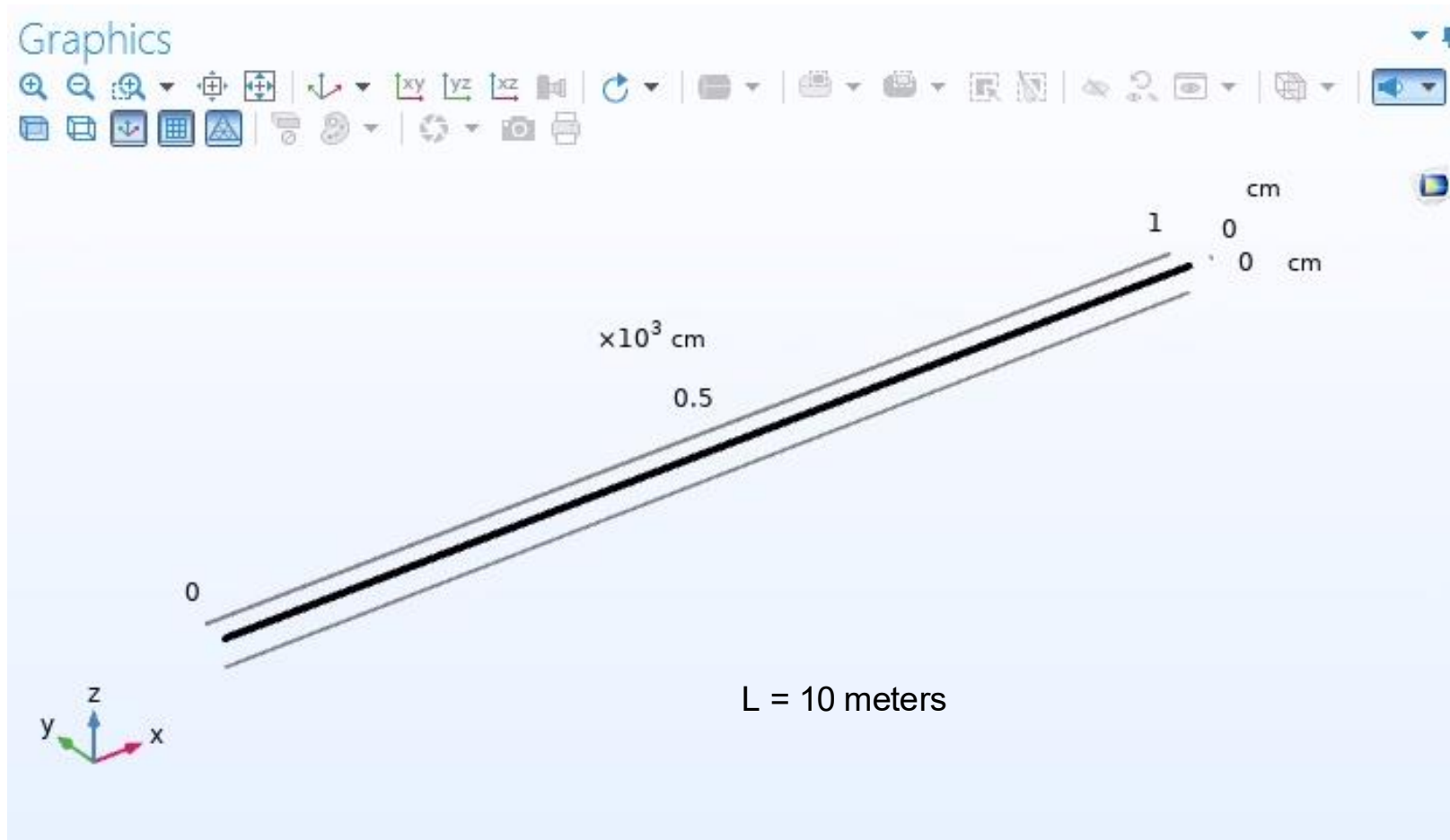
- **Made four cylinders according to diameter parameters**
 - Inner, annulus, shell inner, shell outer
- **Need to separate tube walls and the regions within tubes**
 - Four distinct selections
- **Difference (x4)**
 - Select cylinder you want
 - Select every other cylinder that is not part of that region





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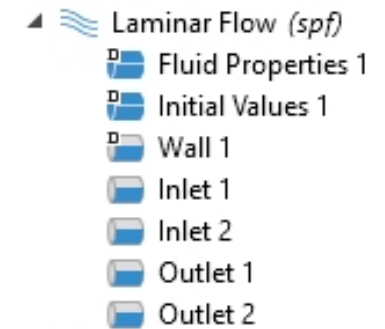
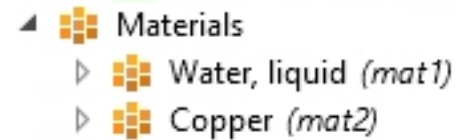
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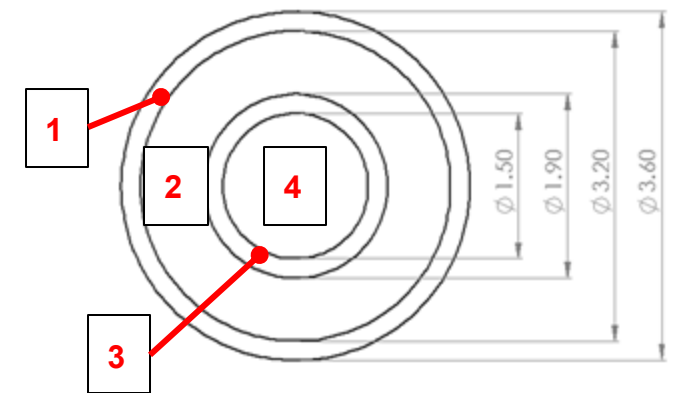
Materials

- **Water**
 - Regions 2 and 4
- **Copper**
 - Regions 1 and 3 (tube walls)



Laminar Flow

- **Set as wall**
 - Region 1 and 3 (tube walls)
- **Inlet and outlet depending on flow configuration**
 - Region 2 and 4 (repeat for both sides!)





Heat Transfer in Fluids

- Set the temperature at both inlets
 - Parallel flow and counterflow only change which sides the inputs and outputs are on
- Inflow and outflow
 - One on each side
- Hot water input, $T_{h,in} = 100^{\circ}\text{C}$
- Cold water input, $T_{c,in} = 20^{\circ}\text{C}$

Name	Expression	Value	Description
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Heat Transfer in Fluids (ht)

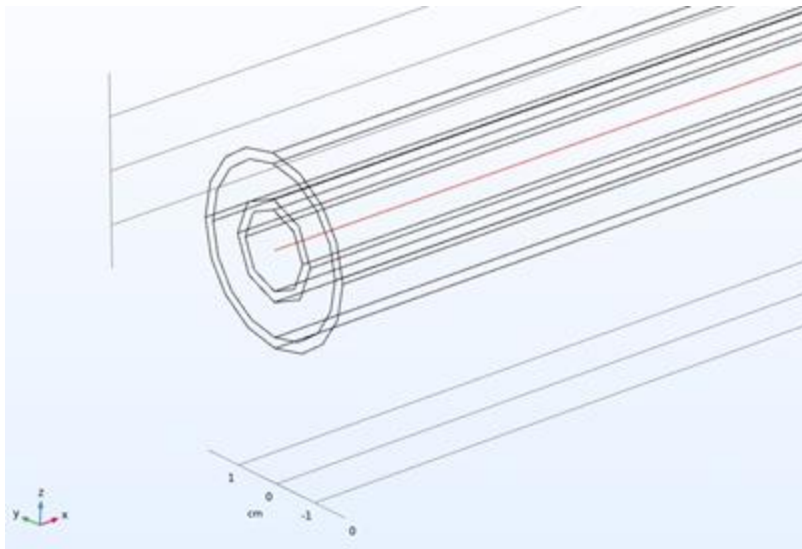
- Fluid 1
- Initial Values 1
- Thermal Insulation 1
- Solid 1
- Inflow 1
- Inflow 2
- Outflow 1
- Outflow 2



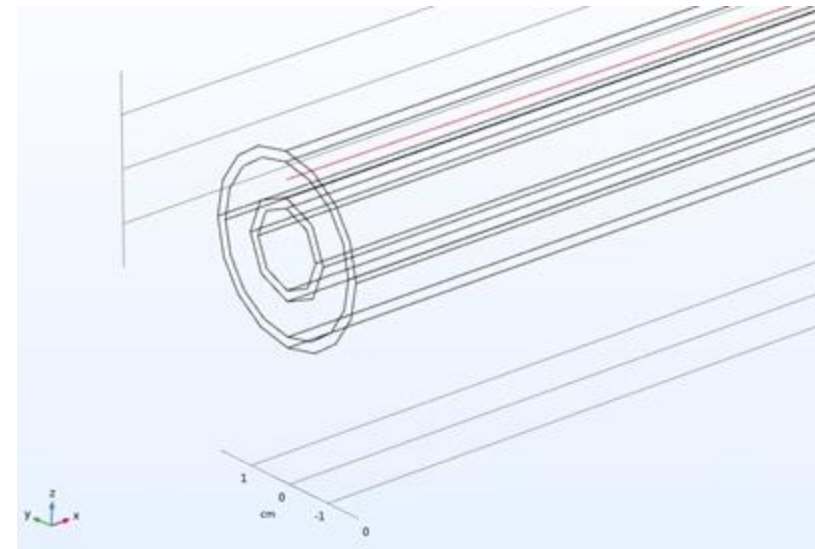
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Two Cut Lines



Hot water cut line

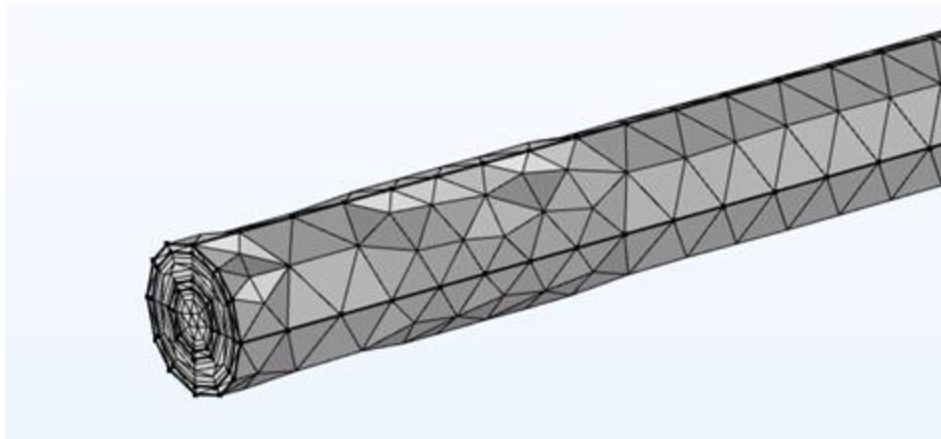


Cold water cut line



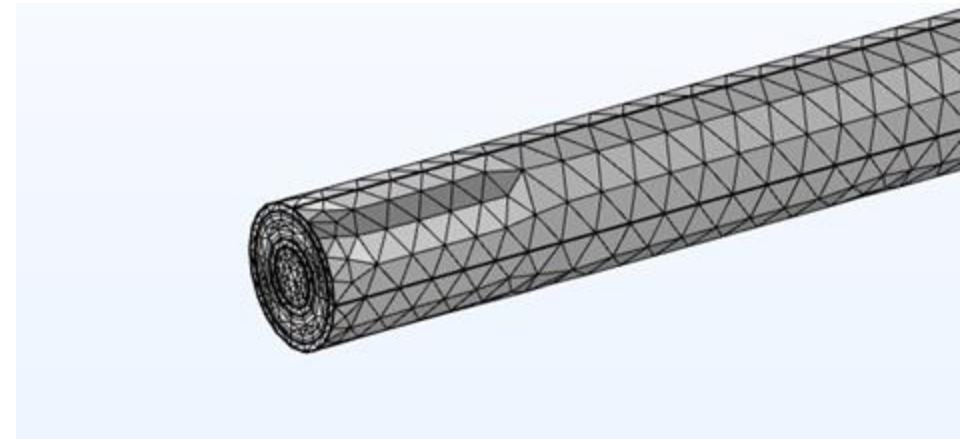
Extra Coarse Mesh

- **754,678 degrees of freedom solved for**
 - Plus 136,936 internal DOFs
- **Computation time**
 - Parallel: 10 minutes, 44 seconds
 - Counterflow: 16 minutes, 15 seconds



Normal Mesh

- **2,096,525 degrees of freedom solved for**
 - Plus 299,186 internal DOFs
- **Computation time**
 - Parallel: 54% after 1 hour
 - Counterflow: 41% after 1 hour





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Expected Results

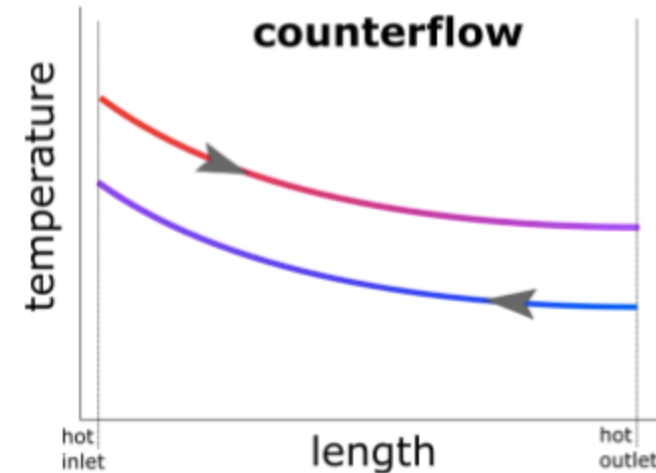
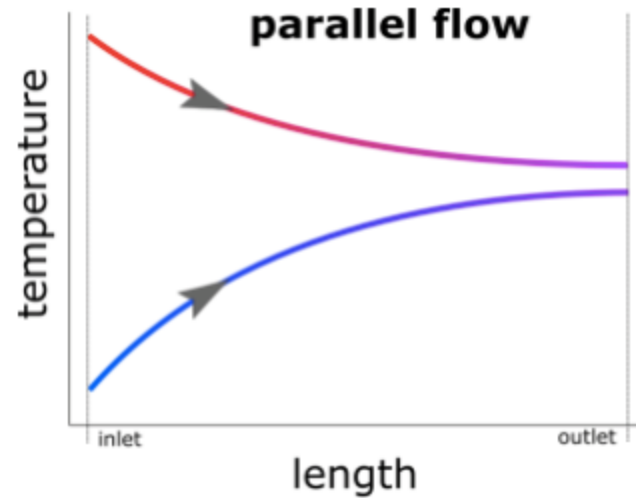


Image from <https://www.wattco.com/2021/06/heat-exchanger-configurations/>



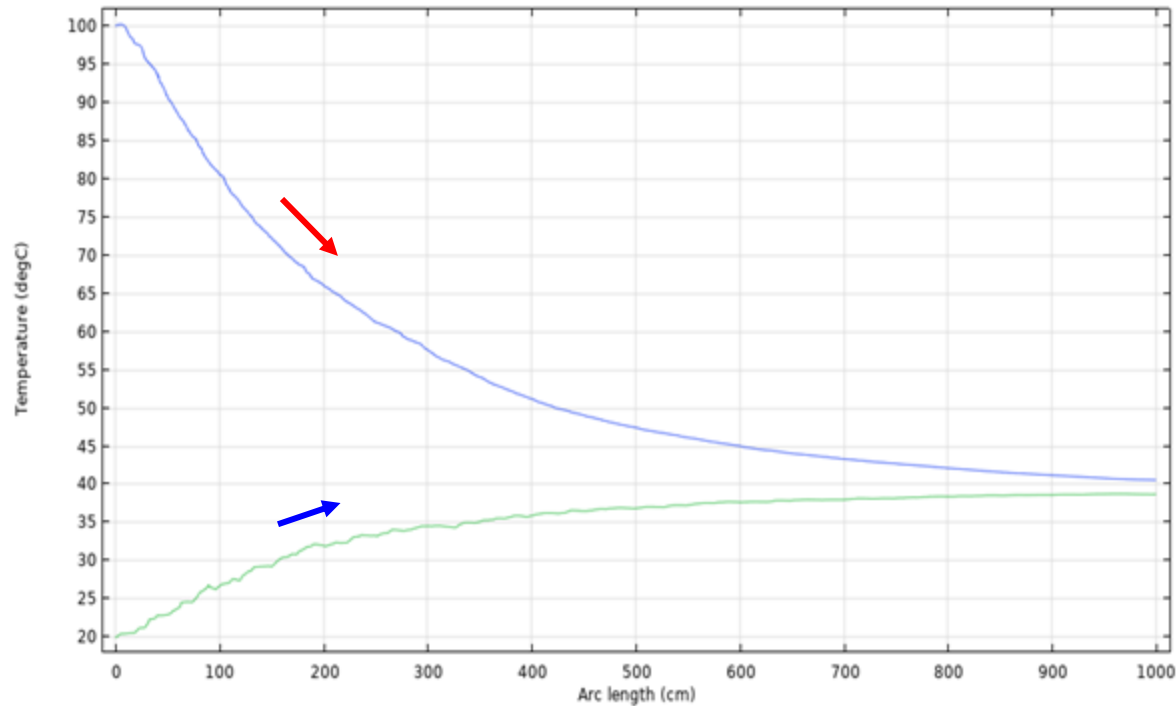
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Extra Coarse Mesh

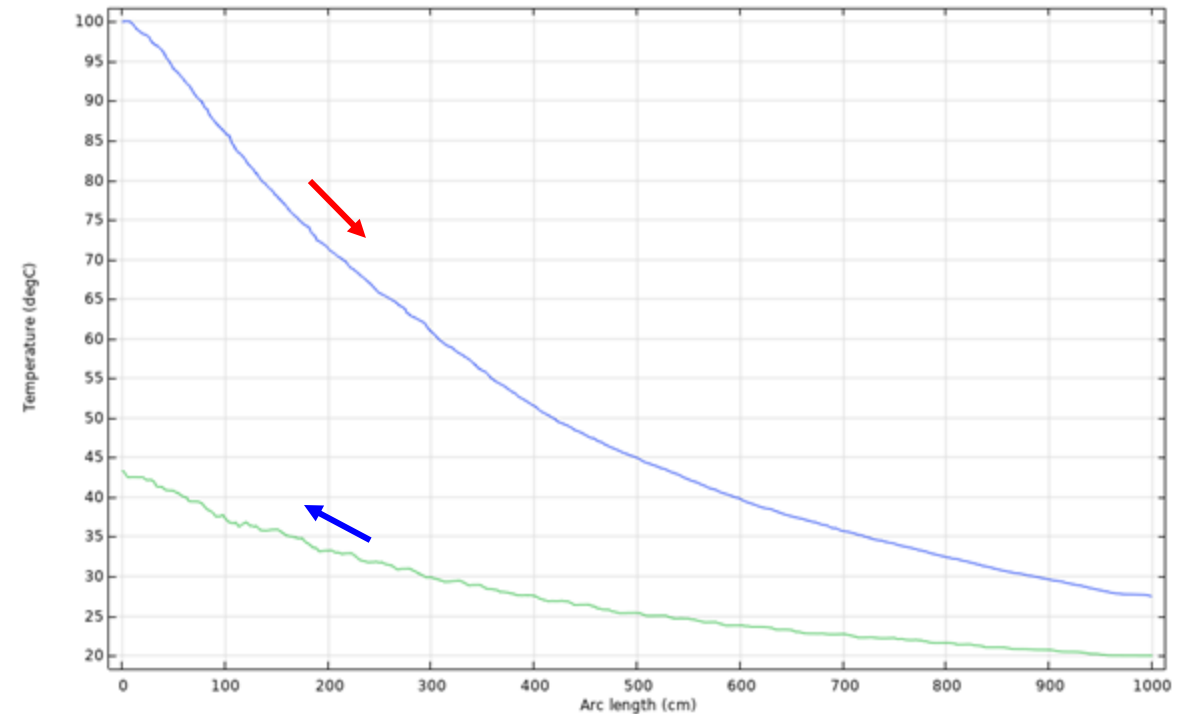
Parallel

Parallel Concentric Heat Exchanger



Counterflow

Counterflow Concentric Heat Exchanger





Parallel

- Hot water input, $T_{h,in} = 100C$
- Hot water output, $T_{h,out} = 41C$
- Cold water input, $T_{c,in} = 20C$
- Cold water output, $T_{c,out} = 38C$

$$\Delta T_h = 59C$$

$$\Delta T_c = 18C$$

Counterflow

- Hot water input, $T_{h,in} = 100C$
- Hot water output, $T_{h,out} = 27C$
- Cold water input, $T_{c,in} = 20C$
- Cold water output, $T_{c,out} = 44C$

$$\Delta T_h = 73C$$

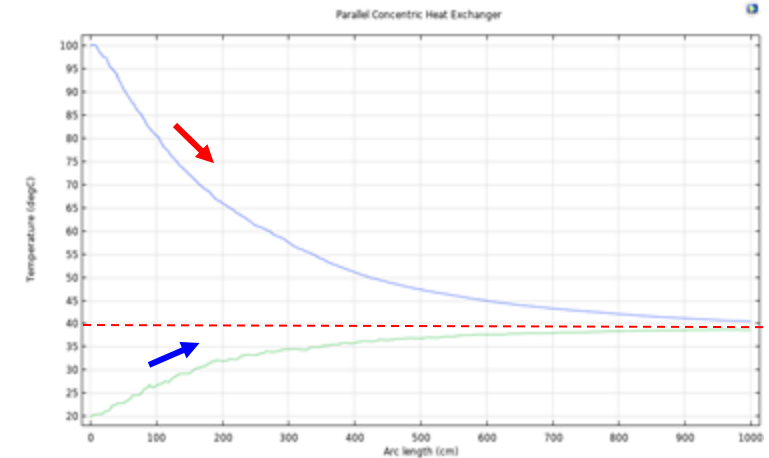
$$\Delta T_c = 24C$$



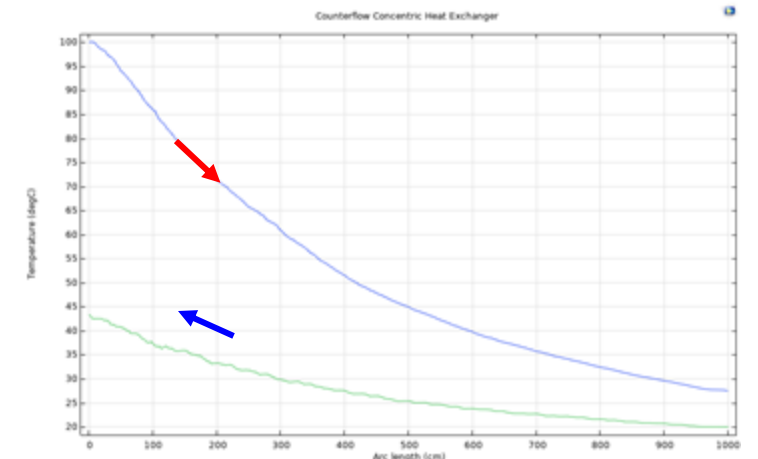
Results

- The counterflow heat exchanger led to a greater change in temperature
- For parallel flow
 - The exit temperature of the cold water did not exceed the exit temperature of the hot water
- For counter flow
 - The hot water got to 27C, close to the cold input
- Having a finer mesh was not worth the computation time
 - The extra coarse mesh gives a decent solution

Parallel



Counterflow





One-Way Coupling Approach

- **A way to save computation time and resources**
- **For nonisothermal flow**
 - Compute the flow field first
 - Use it as an input for the heat transfer problem
- **Other possible multiphysics cases**
 - Turbulent regimes and flow in porous media
 - Advected field (if the coupling is weak)
 - Coupling between heat transfer and structural mechanics leading to thermal expansion

Ref: <https://www.comsol.com/blogs/how-to-save-computational-time-with-a-one-way-coupling-approach/>