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Analysis of a Parallel and Counterflow Concentric Heat Exchanger

MECH 257 Presentation

Presented by Andrew Lemus



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Goal

Model and simulate a parallel and counterflow concentric heat exchanger.

Additionally:

1. Learn how to analyze a thermal+fluid model in COMSOL
2. Consider mesh sizes along with computation time
3. Compare the results of the two heat exchangers

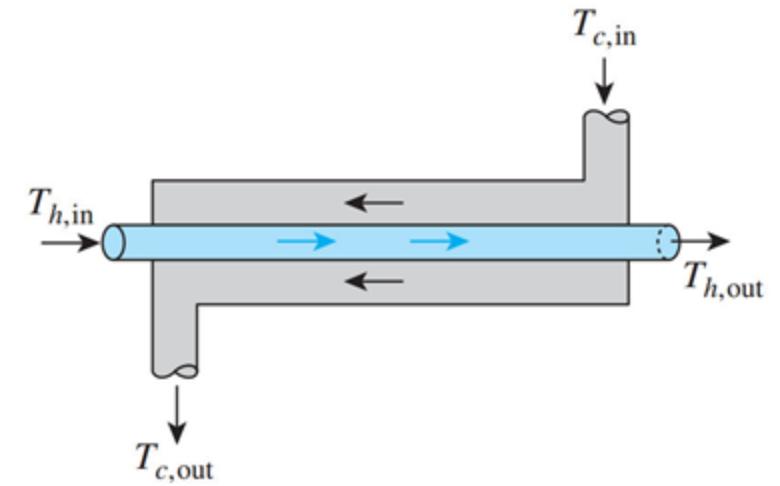


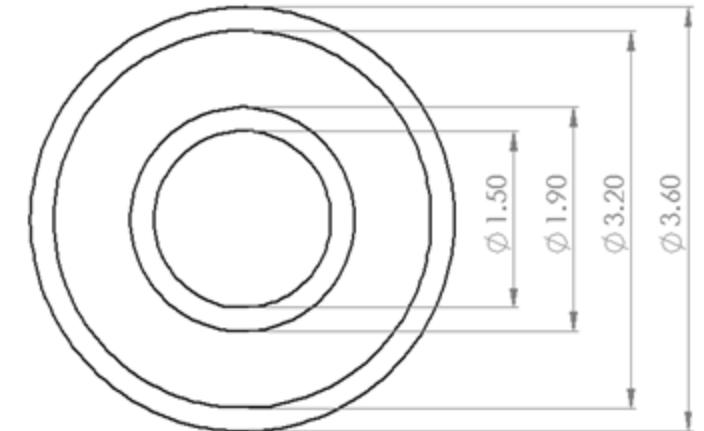
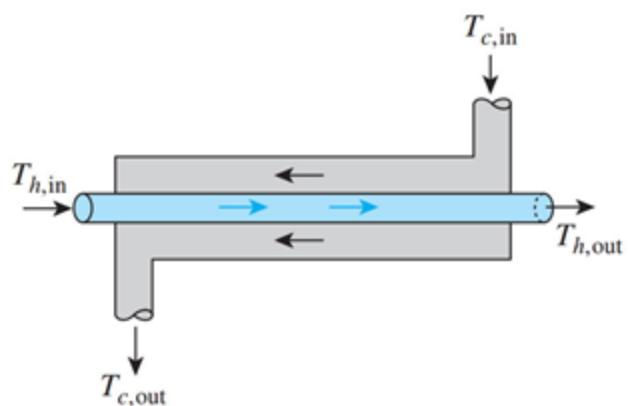
Image from <https://www.thomasnet.com/articles/process-equipment/double-pipe-heat-exchangers/>



Parameters

- Length, $L = 10$ meters
- Hot and cold flow, $U = 1$ m/s
- Hot water input, $T_{h,in} = 100$ C
- Cold water input, $T_{c,in} = 20$ C
- The shell thickness does not matter for this simulation
 - Convection can be applied to outside if desired

Name	Expression	Value	Description
D_i	$1.5[\text{cm}]$	0.015 m	
D_a	$1.9[\text{cm}]$	0.019 m	
$D_{\text{shell_i}}$	$3.2[\text{cm}]$	0.032 m	
$D_{\text{shell_o}}$	$3.6[\text{cm}]$	0.036 m	
L	$10[\text{m}]$	10 m	
$U_{\text{cold_in}}$	$1[\text{m/s}]$	1 m/s	
$U_{\text{hot_in}}$	$1[\text{m/s}]$	1 m/s	
$T_{\text{hot_in}}$	$100[\text{degC}]$	373.15 K	
$T_{\text{cold_in}}$	$20[\text{degC}]$	293.15 K	

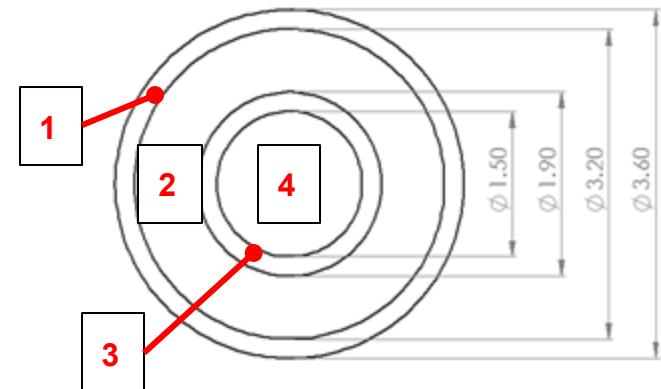




Geometries

- Made four cylinders according to diameter parameters
 - Inner, annulus, shell inner, shell outer
- Need to separate tube walls and the regions within tubes
 - Four distinct selections
- Difference (x4)
 - Select cylinder you want
 - Select every other cylinder that is not part of that region

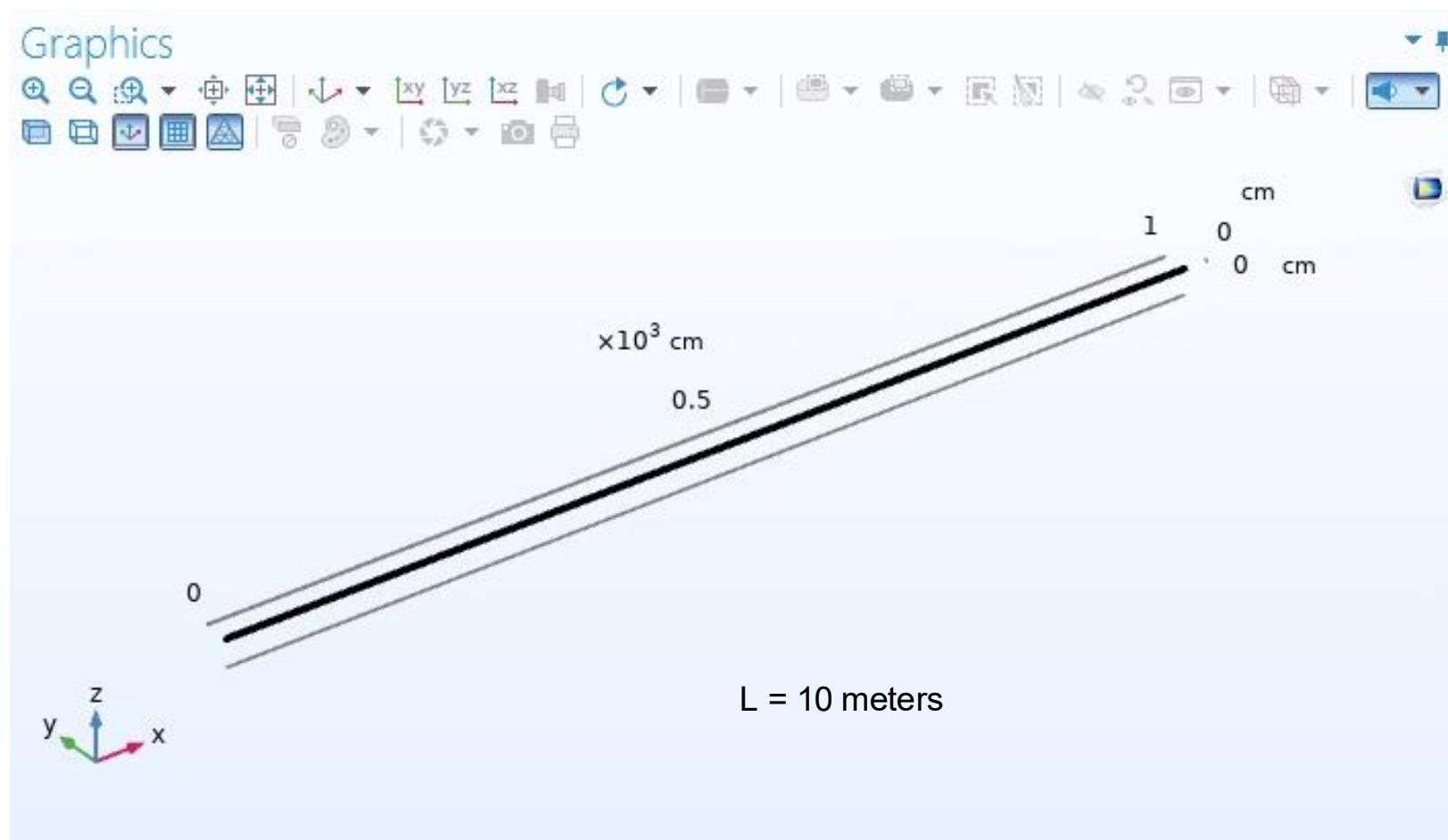
- ◀ Component 1 (comp1)
- ▷ Definitions
- ◀ Geometry 1
 - Inner (cyl1)
 - Annulus (cyl2)
 - Shell Inner (cyl3)
 - Shell Outer (cyl4)
 - Difference 2 (dif2)
 - Difference 1 (dif1)
 - Difference 3 (dif3)
 - Difference 4 (dif4)
 - Flow Area (uni1)
 - Tubes (uni2)
 - Compose 1 (co1)
 - Form Union (fin)





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Materials

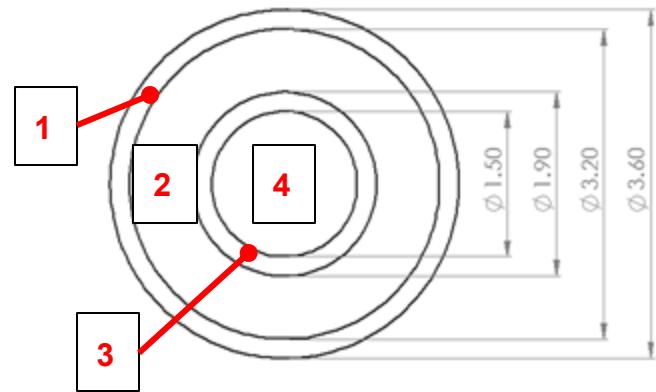
- Water
 - Regions 2 and 4
- Copper
 - Regions 1 and 3 (tube walls)

- ▲ Materials
 - ▷ Water, liquid (mat1)
 - ▷ Copper (mat2)

- ▲ Laminar Flow (spf)
 - ▷ Fluid Properties 1
 - ▷ Initial Values 1
 - ▷ Wall 1
 - ▷ Inlet 1
 - ▷ Inlet 2
 - ▷ Outlet 1
 - ▷ Outlet 2

Laminar Flow

- Set as wall
 - Region 1 and 3 (tube walls)
- Inlet and outlet depending on flow configuration
 - Region 2 and 4 (repeat for both sides!)

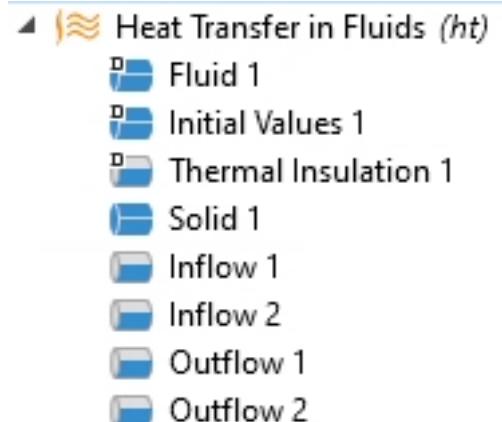




Heat Transfer in Fluids

- Set the temperature at both inlets
 - Parallel flow and counterflow only change which sides the inputs and outputs are on
- Inflow and outflow
 - One on each side
- Hot water input, $T_{h,in} = 100C$
- Cold water input, $T_{c,in} = 20C$

Name	Expression	Value	Description
Di	1.5[cm]	0.015 m	
Da	1.9[cm]	0.019 m	
Dshell_i	3.2[cm]	0.032 m	
Dshell_o	3.6[cm]	0.036 m	
L	10[m]	10 m	
Ucold_in	1[m/s]	1 m/s	
Uhot_in	1[m/s]	1 m/s	
Thot_in	100[degC]	373.15 K	
Tcold_in	20[degC]	293.15 K	

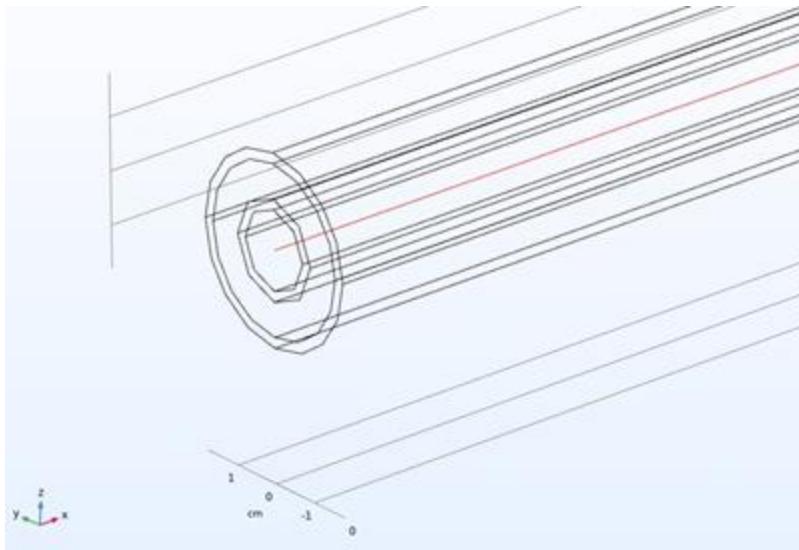




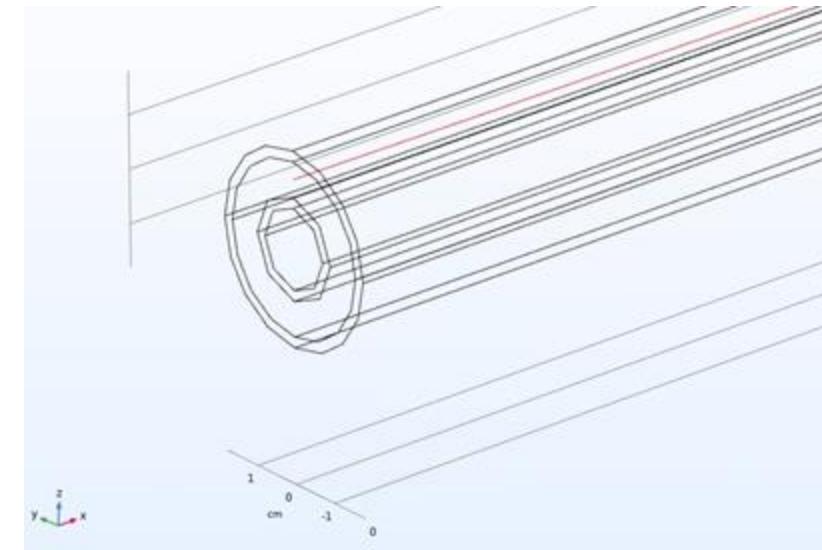
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Two Cut Lines



Hot water cut line

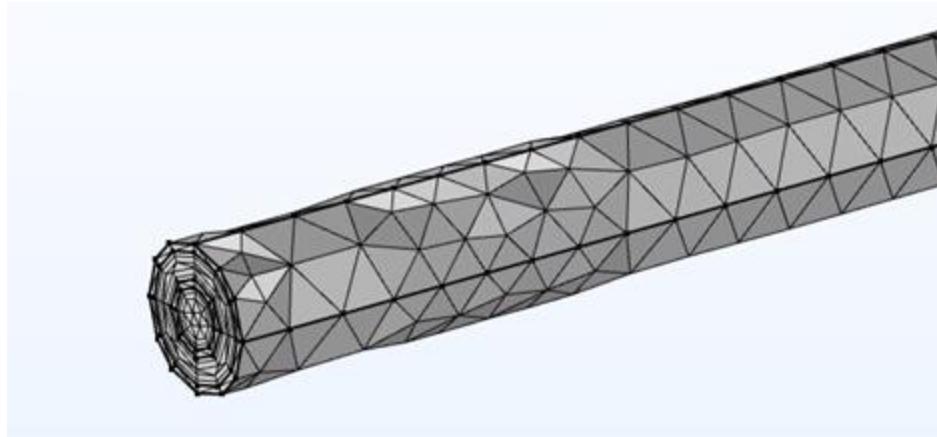


Cold water cut line



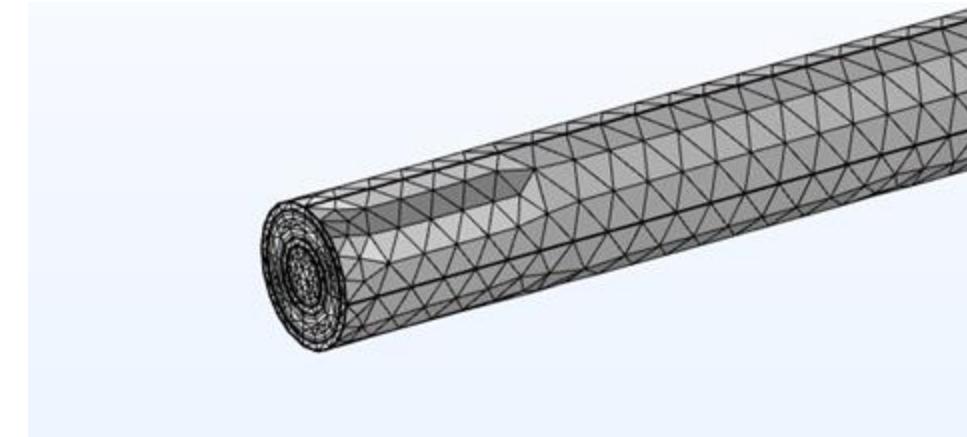
Extra Coarse Mesh

- **754,678 degrees of freedom solved for**
 - Plus 136,936 internal DOFs
- **Computation time**
 - Parallel: 10 minutes, 44 seconds
 - Counterflow: 16 minutes, 15 seconds



Normal Mesh

- **2,096,525 degrees of freedom solved for**
 - Plus 299,186 internal DOFs
- **Computation time**
 - Parallel: 54% after 1 hour
 - Counterflow: 41% after 1 hour





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Expected Results

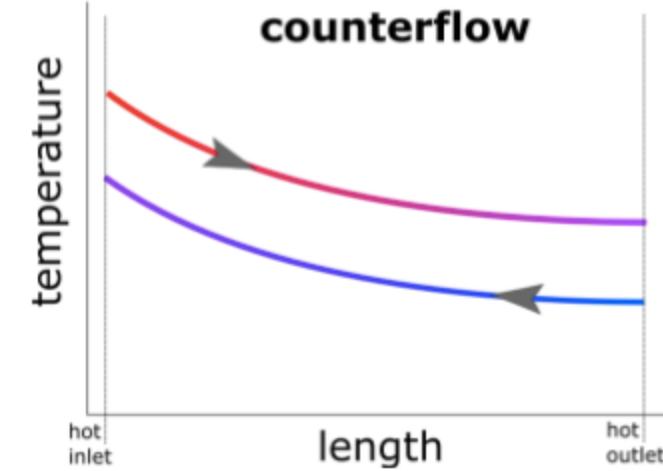
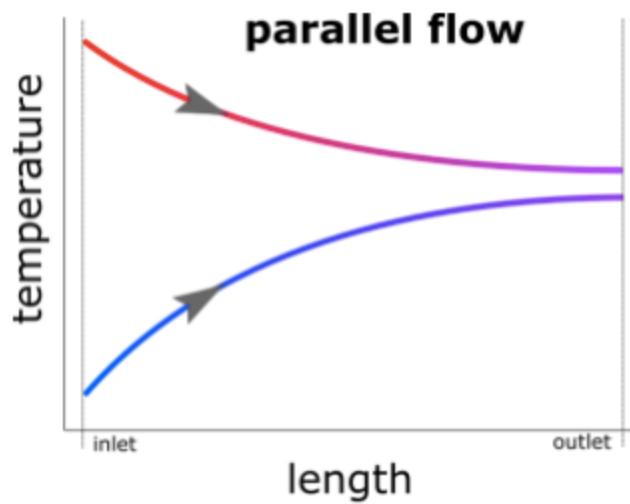


Image from <https://www.wattco.com/2021/06/heat-exchanger-configurations/>



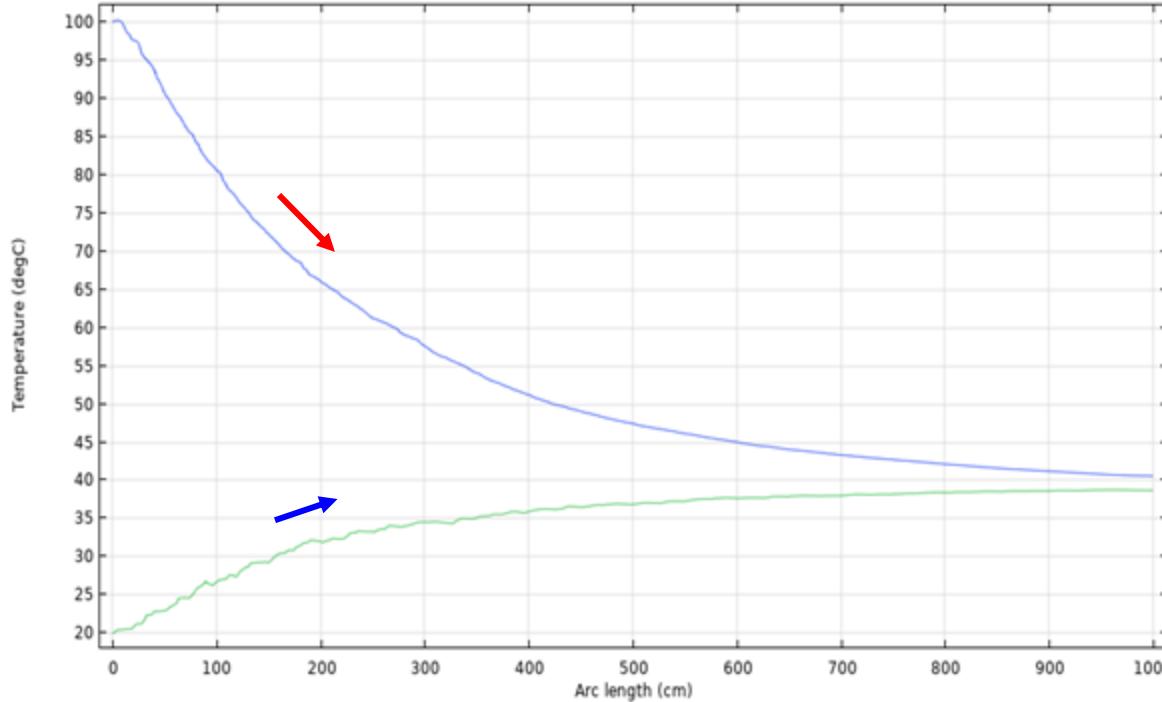
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Extra Coarse Mesh

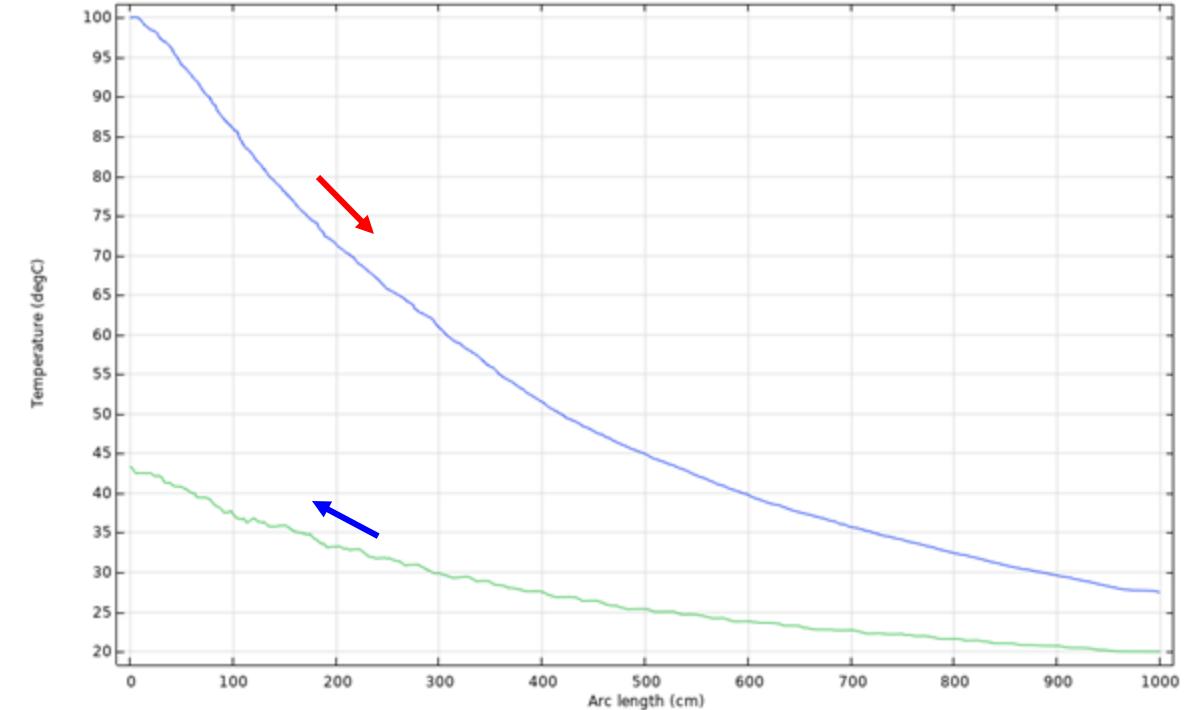
Parallel

Parallel Concentric Heat Exchanger



Counterflow

Counterflow Concentric Heat Exchanger



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Parallel

- Hot water input, $T_{h,in} = 100C$
- Hot water output, $T_{h,out} = 41C$
- Cold water input, $T_{c,in} = 20C$
- Cold water output, $T_{c,out} = 38C$

$$\Delta T_h = 59C$$

$$\Delta T_c = 18C$$

Counterflow

- Hot water input, $T_{h,in} = 100C$
- Hot water output, $T_{h,out} = 27C$
- Cold water input, $T_{c,in} = 20C$
- Cold water output, $T_{c,out} = 44C$

$$\Delta T_h = 73C$$

$$\Delta T_c = 24C$$

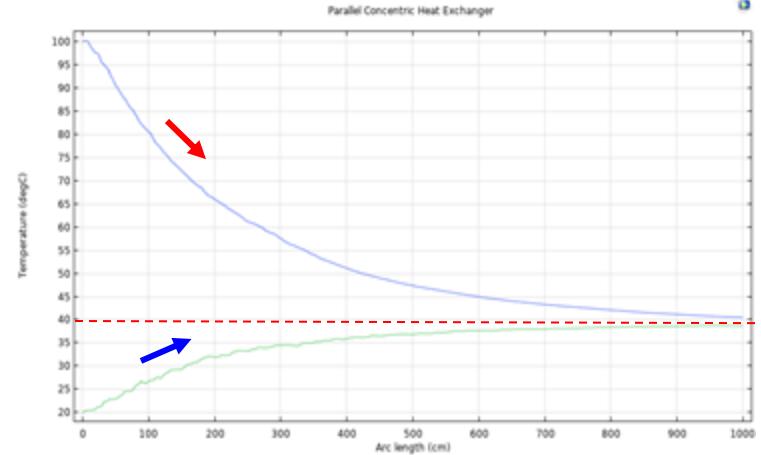




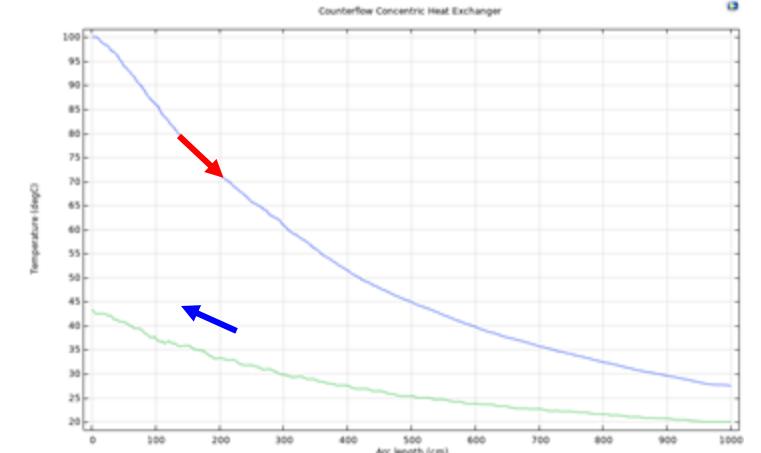
Results

- The counterflow heat exchanger led to a greater change in temperature
- For parallel flow
 - The exit temperature of the cold water did not exceed the exit temperature of the hot water
- For counter flow
 - The hot water got to 27C, close to the cold input
- Having a finer mesh was not worth the computation time
 - The extra coarse mesh gives a decent solution

Parallel



Counterflow





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One-Way Coupling Approach

- A way to save computation time and resources
- For nonisothermal flow
 - Compute the flow field first
 - Use it as an input for the heat transfer problem
- Other possible multiphysics cases
 - Turbulent regimes and flow in porous media
 - Advecting field (if the coupling is weak)
 - Coupling between heat transfer and structural mechanics leading to thermal expansion

Ref: <https://www.comsol.com/blogs/how-to-save-computational-time-with-a-one-way-coupling-approach/>