



An efficient methodology based on 2.5D FEM-BEM for ground-borne vibration radiated by underground railway tunnels and the re-radiated noise emitted inside them

Doctoral Dissertation Presentation

Dhananjay Ghangale

Advisors:

Dr. Robert Arcos

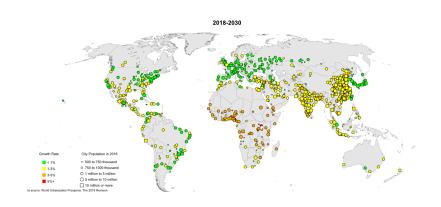
Dr. Jordi Romeu

Overview

- 1. Introduction
- 2. Novelties proposed in 2.5D FEM-BEM
- 3. Accurate modelling of train response
- 4. Vibration energy flow
- 5. Re-radiated noise
- 6. Conclusions

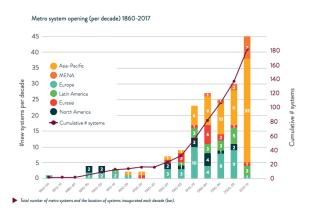
Introduction

Global outlook





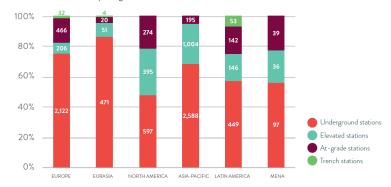
Global outlook



UITP Observatory of Automated Metros

Global outlook

Metro construction models per region



Distribution of construction models for metro stations, according to world region.

Metro in News

mylondon (Europe-UK) London family's nightmare as Tube trains make the walls rattle like thunder

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Vibrations caused by Delhi Metro's underground trains creating cracks in buildings

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Metro Reasons (America-USA)

A new Metro study confirms train vibrations, but has little recourse for residents

· Models for SSI analysis:

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 - 1. Empirical

- · Models for SSI analysis:
 - 1. Empirical
 - 2. Semi-analytical

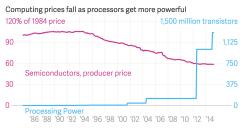
- · Models for SSI analysis:
 - 1. Empirical
 - 2. Semi-analytical
 - 3. Numerical

Method	Modelling detail	Speed	Computational resources
Numerical	high	low	high
Semi-analytical	low	high	low

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△ T L △ S Data: US Bureau of Labor Statistics, FactSet, International Control of Con

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 - 3. Numerical
 - 2D
 - · 3D



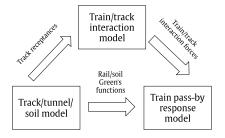
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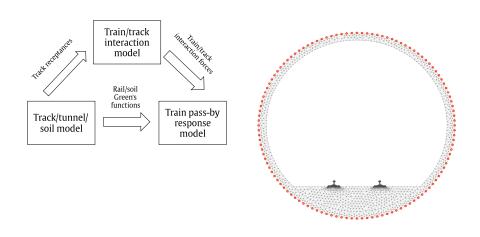
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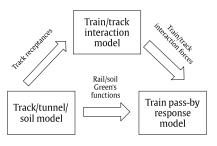
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 - · 3D
 - · 2.5D



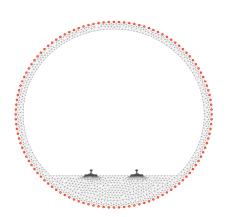
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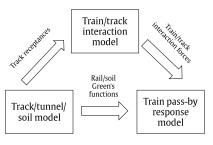






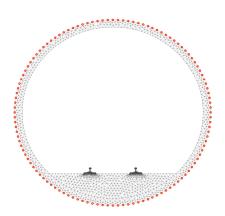
Track/tunnel/soil modelling

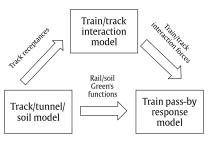




Track/tunnel/soil modelling

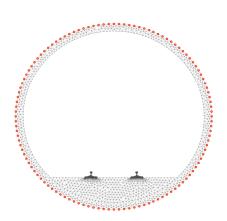
· 2.5D FEM-PML

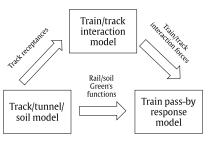




Track/tunnel/soil modelling

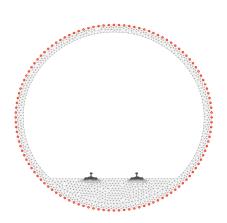
- · 2.5D FEM-PML
- · 2.5D FEM-MFS





Track/tunnel/soil modelling

- · 2.5D FEM-PML
- 2.5D FEM-MFS
- 2.5D FEM-BEM



Coupled elastodynamic FEM-BEM modelling approach based on 2.5D methodology.

^{*}S. François et al. (2010), *M Bonnet (1995)

Coupled elastodynamic FEM-BEM modelling approach based on 2.5D methodology.

1. 2.5D FEM-BEM based on globally regularised singular integrals.*

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- 1. 2.5D FEM-BEM based on globally regularised singular integrals.*
- 2. Response is obtained by constructing a global stiffness matrix

$$[\mathbf{K}_0 - \mathrm{i} k_x \mathbf{K}_1 + k_x^2 \mathbf{K}_2 + \overline{\mathbf{K}}_s - \omega^2 \mathbf{M}] \overline{\mathbf{U}} = \overline{\mathbf{F}}.$$

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3. 2.5D FEM-BEM is extensively verified.

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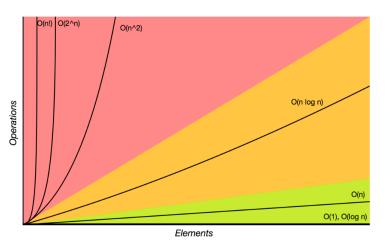
$$[\mathbf{K}_0 - \mathrm{i} k_{\mathsf{X}} \mathbf{K}_1 + k_{\mathsf{X}}^2 \mathbf{K}_2 + \mathbf{\bar{K}}_{\mathsf{S}} - \omega^2 \mathbf{M}] \mathbf{\bar{U}} = \mathbf{\bar{F}}.$$

- 3. 2.5D FEM-BEM is extensively verified.
- 4. Soil is modelled as full-space.

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Novelties proposed in 2.5D FEM-BEM

Order Complexity: $O(n \log n)$

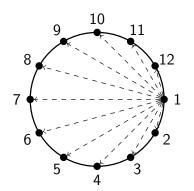


courtsey: bigocheatsheet.com

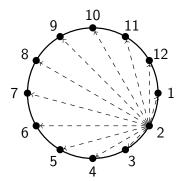
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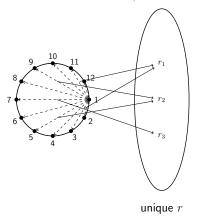


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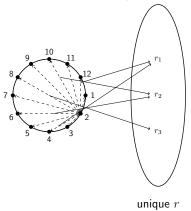
• Setting z = 0 in fullspace Green's functions allows: $F_G = f(\theta)f(r)$.

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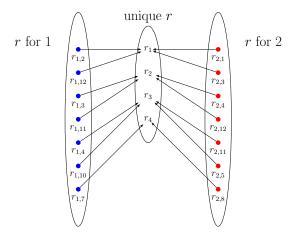
Fast Computation of Green's Function

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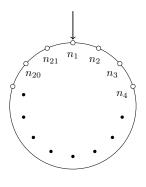
 $r \xrightarrow{unique} r_u$ compute: $Fg \forall r_u$ then construct: $F_G = f(\theta)\mathcal{M}(F_g)$.

- Computation time \propto number of source/receiver points.
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- · Green's displacements:

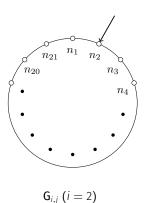
$$\bar{\mathbf{H}} = \mathbf{T}_{\theta}^{-1} \mathcal{M}(\bar{\mathbf{H}}_{us}) \mathbf{T}_{\theta}$$

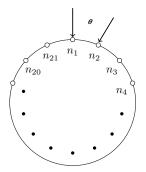
Green's tractions:

$$\overline{T} = T_{\theta}^{-1}(\mathcal{M}(\overline{T}_{us}) \circ T_{\phi})T_{\theta}.$$

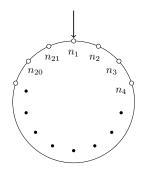


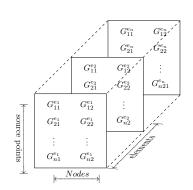
 $G_{i,j}$ (i = 1)





 $G_{2,i} \simeq G_{1,i}$ except for relative rotation between i = 1 and i = 2.





$$G_{i,j}$$
 ($\forall i \in \{2, n\}$) = $\mathcal{M}(\mathsf{T}_{\theta_{rel}}^{-1}\mathsf{G}_{1,j}\mathsf{T}_{\theta_{rel}})$

Accurate modelling of train response

• Two components of the loads are considered:

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 - 2. Dynamic load

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$$F^{w/r} = \left(H_v^{w/r} + H_r^{w/r} + k_H^{-1}I\right)^{-1} E_r.$$

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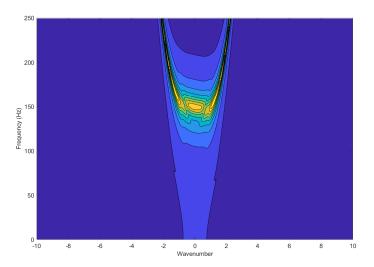
· Displacement obtained after train pass:

$$u(\tilde{x},t) = \frac{1}{4\pi^2} \int_{-\infty}^{+\infty} \sum_{n=1}^{N_a} \left[\int_{-\infty}^{+\infty} \tilde{H}_s^r F_n^{w/r} e^{-ik_x(\tilde{x}-\tilde{x}_n)} dk_x \right] e^{i\tilde{\omega}t} d\tilde{\omega}.$$

Wavenumber-frequency sampling

 \cdot 3D response of train pass-by \propto the Green's functions of the system sampled in 2.5D.

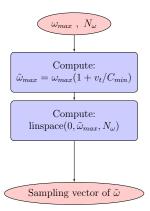
Wavenumber-frequency sampling

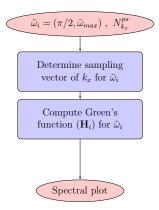


Wavenumber-frequency sampling

- To accurately compute the train pass-by in 2.5D a sampling strategy:
 - 1. Linear on moving frequency.
 - 2. Non-uniform on wavenumber which varies with frequency.

Frequency sampling





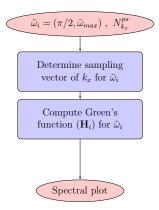
Determining the pre-sampling vector of k_x for $\tilde{\omega}_i$:

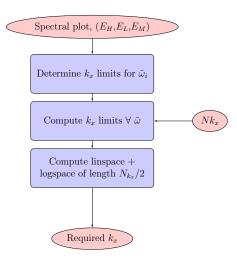
• Compute limits for k_x as:

$$R_{X_i}^{\lim} = \tilde{\omega}/c_i \ \forall \ N_{SS}$$

• Presampling k_x vector:

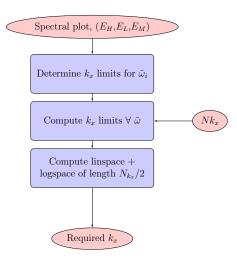
linspace(
$$-2R_{xi}^{\text{lim}}$$
, $2R_{xi}^{\text{lim}}$, $(N_{R_x}^{ps}/2N_{ss})$)
 $\pm \text{logspace}(2R_{xi}^{\text{lim}}$, 10^5 , $(N_{R_x}^{ps}/4N_{ss})$)





Determining the wavenumber limits for $\tilde{\omega}_i$:

- 10% tolerance for linear sampling \implies 90% spectral content of Green's function.
- 1% tolerance for logarithmic sampling \implies 90 \rightarrow 99% spectral content of Green's function.



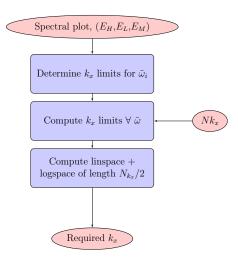
Determining the wavenumber limits for all $\tilde{\omega}$:

· Wavenumber limits vary linearly with frequency

$$k_{\chi_{\rm lin}}^{\rm lim}(\tilde{\omega}) = \left(\frac{\tilde{\omega} - \pi/2}{\tilde{\omega}_{\rm max} - \pi/2}\right) \left(k_{\chi_{\rm lin}}^{\rm lim_{\it u}} - k_{\chi_{\rm lin}}^{\rm lim_{\it d}}\right) + k_{\chi_{\rm lin}}^{\rm lim_{\it u}}$$

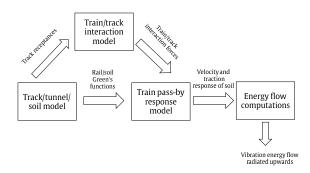
and

$$k_{x_{\rm log}}^{\rm lim}(\tilde{\omega}) = \left(\frac{\tilde{\omega} - \pi/2}{\tilde{\omega}_{\rm max} - \pi/2}\right) \left(k_{x_{\rm log}}^{\rm lim_{\it u}} - k_{x_{\rm log}}^{\rm lim_{\it d}}\right) + k_{x_{\rm log}}^{\rm lim_{\it u}}$$

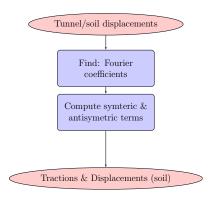


Vibration energy flow

Vibration Energy-Flow



Model for the vibration propagation in the soil

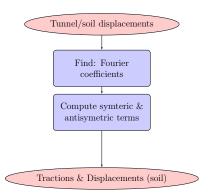


Model for the vibration propagation in the soil

From the displacement in tunnel/soil interface unknown Fourier Coefficients \bar{C}_n^i , (i=s/a) are found by inverting:

$$\bar{\mathbf{U}}_n^i(r_b) = \bar{\mathbf{U}}_{cn}^i(r_b)\bar{\mathbf{C}}_n^i.$$

Model for the vibration propagation in the soil



After which the symmetric and anti-symmetric terms for any radial location in soil r_f are found

$$\bar{\mathbf{U}}_n^i(r_f) = \bar{\mathbf{U}}_{cn}^i(r_f)\bar{\mathbf{C}}_n^i,$$

and

$$\bar{\mathbf{T}}_n^i(r_f) = \bar{\mathbf{T}}_{cn}^i(r_f)\bar{\mathbf{C}}_n^i.$$

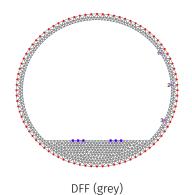
The displacements and tractions in soil at any radial location in soil r_f are given by

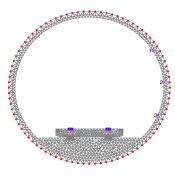
$$\bar{U}_s(r_f,\theta) = \sum_{n=0}^{\infty} [S_n^s(\theta)\bar{U}_n^s(r_f) + S_n^a(\theta)\bar{U}_n^a(r_f)].$$

and

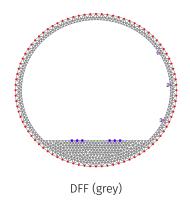
$$\overline{T}_{s}(r_{f},\theta) = \sum_{n=0}^{\infty} [S_{n}^{s}(\theta)\overline{T}_{n}^{s}(r_{f}) + S_{n}^{a}(\theta)\overline{T}_{n}^{a}(r_{f})].$$

Forrest and Hunt et al. (2006), Clot (2014)

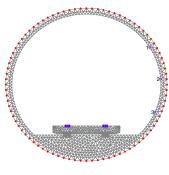




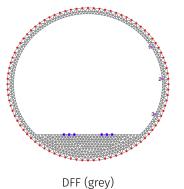
FST (black)



Two types of soil are considered:

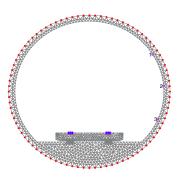


FST (black)

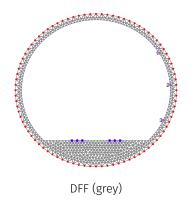


Two types of soil are considered:

- · Soft Soil (b) (E = 180 MPa).
- · Hard Soil (a) (E = 480 MPa).

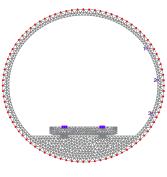


FST (black)



Two types of soil are considered:

- Soft Soil (b)(E = 180 MPa).
- Hard Soil (a)
 (E = 480 MPa).
- 38 evaluators in soil at a specific r.



FST (black)

Vibration energy flow

Energy flow:

$$E = \int_{-\infty}^{\infty} P(t) dt,$$

Power flow is defined as:

$$P(t) = \int_{S} \mathbf{v}(\mathbf{x}, t) \cdot \boldsymbol{\tau}(\mathbf{x}, t) dS,$$

Energy flow:

$$E = \int_{-\infty}^{\infty} P(t) dt,$$

Power flow is defined as:

$$P(t) = \int_{S} \mathbf{v}(\mathbf{x}, t) \cdot \boldsymbol{\tau}(\mathbf{x}, t) dS,$$

 Vibration energy flow radiated upwards in soft soil

Track type	5 m	15 m
DFF	0.2547 J/m	0.1691 J/m
FST	0.2227 J/m	0.1711 J/m

· Energy flow:

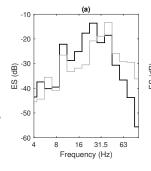
$$E=\int_{-\infty}^{\infty}P(t)\mathrm{d}t,$$

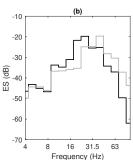
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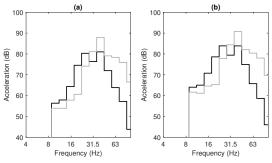
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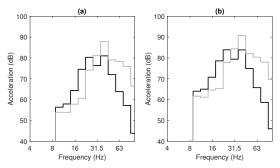


- · (a) 5 m (b) 15 m
- · DFF (grey) FST (black)



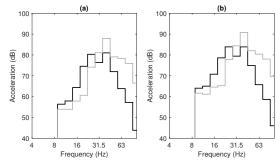
- · DFF (grey) FST (black)
- · Hard soil (a) Soft Soil (b)

 FST is better at reducing vibration at higher frequencies.

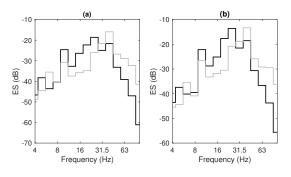


- DFF (grey) FST (black)
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- FST is better at reducing vibration at higher frequencies.
- Level of vibration in soft soil > hard soil.

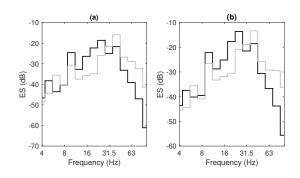


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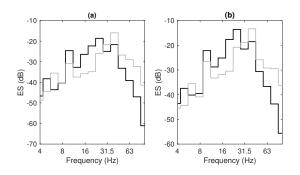
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 FST shifts the energy spectra to lower frequencies.

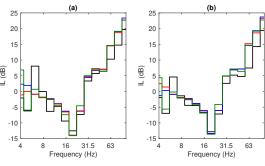


- DFF (grey) FST (black)
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- FST shifts the energy spectra to lower frequencies.
- Local soil affects in levels of induced vibrations and vibration energy radiated.



- · DFF (grey) FST (black)
- · Hard soil (a) Soft Soil (b)



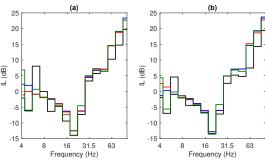
Insertion loss by energy spectral density:

$$\mathrm{IL_{ES}} = 10 \, log_{10} \left(\frac{\mathrm{ES_{FST}}}{\mathrm{ES_{DFF}}} \right);$$

Insertion loss by acceleration:

$$IL_{a} = 20 \log_{10} \left(\frac{a_{FST}}{a_{DFF}} \right).$$

- ESD (black) accleration (colored)
- · Hard soil (a) Soft Soil (b)



Insertion loss by energy spectral density:

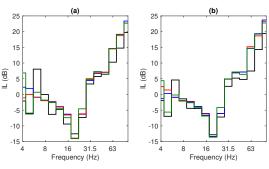
$$\mathrm{IL_{ES}} = 10 \, log_{10} \left(\frac{\mathrm{ES_{FST}}}{\mathrm{ES_{DFF}}} \right);$$

Insertion loss by acceleration:

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· IL_a \simeq IL_{ES}.

- ESD (black) accleration (colored)
- Hard soil (a) Soft Soil (b)



- ESD (black) accleration (colored)
- Hard soil (a) Soft Soil (b)

Insertion loss by energy spectral density:

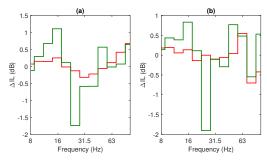
$$\mathrm{IL}_{\mathrm{ES}} = 10 \, log_{10} \left(\frac{\mathrm{ES}_{\mathrm{FST}}}{\mathrm{ES}_{\mathrm{DFF}}} \right) ;$$

Insertion loss by acceleration:

$$\mathrm{IL_{a}} = 20 \, log_{10} \, \bigg(\frac{a_{\mathrm{FST}}}{a_{\mathrm{DFF}}} \bigg).$$

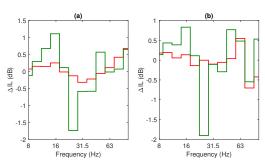
- · IL_a \simeq IL_{ES}.
- Mismatch at high & low frequencies of about 8 dB

Validity of using one accelerometer



· Hard soil (a) Soft Soil (b)

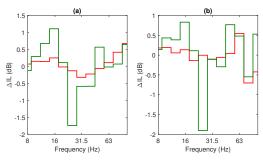
Validity of using one accelerometer



· Hard soil (a) Soft Soil (b)

 IL of a mitigation measure could be significantly dependent on the local subsoil surrounding.

Validity of using one accelerometer

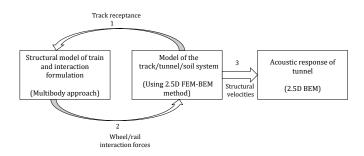


Hard soil (a) Soft Soil (b)

- IL of a mitigation measure could be significantly dependent on the local subsoil surrounding.
- Variations in the IL associated with the vibration acceleration are found to be small ⇒ location of the accelerometer is not of great importance.

Re-radiated noise

Re-radiated Noise



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2.5D Acoustic BEM

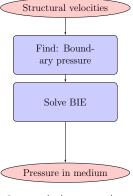
- · Global regularisation is extended to 2.5D Acoustic BEM.
- Two step procedure:
 - 1. Solve BIE to find boundary unknowns:

$$\bar{\mathsf{H}}_b \bar{\mathsf{P}}_{n_b} = \mathrm{i} \rho \omega \bar{\mathsf{G}}_b \bar{\mathsf{V}}_{n_b}.$$

2. Then find pressure in domain:

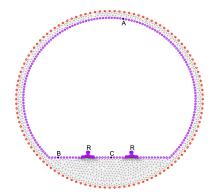
$$\bar{\mathbf{P}}_{n_f} = -(\bar{\mathbf{H}}_f \bar{\mathbf{P}}_{n_b} + \mathrm{i} \rho \omega \bar{\mathbf{G}}_f \bar{\mathbf{V}}_{n_b}).$$

2.5D Acoustic BEM

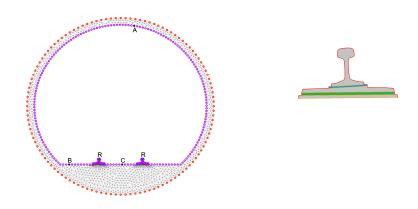


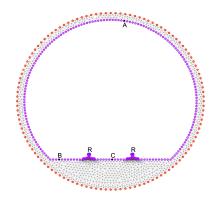
Decoupled approach.

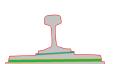
Vibration and re-radiated noise inside underground tunnels



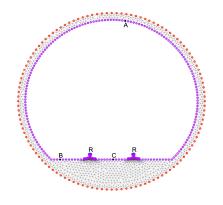
Vibration and re-radiated noise inside underground tunnels





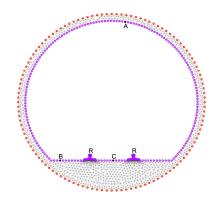


· Case 1: Stiffer rail pad (black) $(E_t = 1.15 \text{ MPa}), (E_b = 2.7 \text{ MPa})$





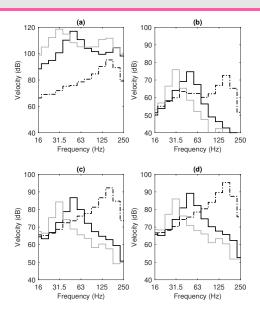
- Case 1: Stiffer rail pad (black) $(E_t = 1.15 \text{ MPa}), (E_b = 2.7 \text{ MPa})$
- Case 2: Softer rail pad (grey) $(E_t = 0.3 \text{ MPa}), (E_b = 0.7 \text{ MPa})$



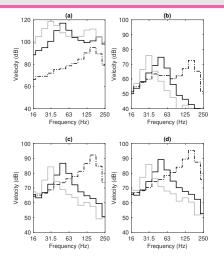


- Case 1: Stiffer rail pad (black) $(E_t = 1.15 \text{ MPa}), (E_b = 2.7 \text{ MPa})$
- Case 2: Softer rail pad (grey) $(E_t = 0.3 \text{ MPa}), (E_b = 0.7 \text{ MPa})$
- Case 3: No rail pads (dashed)

Influence of fastener stiffness on vibrations inside underground tunnels



Influence of fastener stiffness on vibrations inside underground tunnels

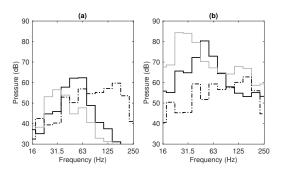


rail response (a) tunnel response (b,c,d)

- Softer rail pads ⇒
 more rail vibration and
 less vibration from tunnel.
- No rail pads

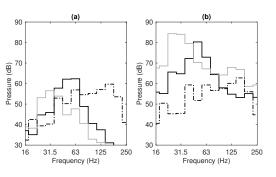
 rail and tunnel structure have identical vibration radiation.

Influence of fastener stiffness on re-radiated noise inside underground tunnels



- · case 1 (black) case 2 (grey) case 3 (dashed)
- noise levels by tunnel (a)
 noise levels by rail + tunnel (b)

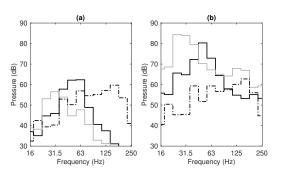
Influence of fastener stiffness on re-radiated noise inside underground tunnels



 Total noise pressure level is influenced by rails.

- · case 1 (black) case 2 (grey) case 3 (dashed)
- noise levels by tunnel (a)
 noise levels by rail + tunnel (b)

Influence of fastener stiffness on re-radiated noise inside underground tunnels



- · case 1 (black) case 2 (grey) case 3 (dashed)
- noise levels by tunnel (a)
 noise levels by rail + tunnel (b)

- Total noise pressure level is influenced by rails.
- Softer fastener increases total noise pressure levels inside the tunnel

Conclusions

- 1. 2.5D FEM-BEM for elastodynamics
 - · Fast Computation of Green's functions.
 - Axisymmetric approach for underground tunnel.
- 2. 2.5D BEM for acoustics based on globally regularised integrals.
- Methodology for computing vibration energy flow by underground railway infrastructures.
- 4. Methodology for computing re-radiated noise.
- 5. New non-uniform wavenumber sampling for train response.

Conclusion from vibration energy flow study:

Conclusion from vibration energy flow study:

- 1. FST is reducing drastically the levels of vibration as compared with the DFF system at higher frequencies.
- 2. FST shifts the vibration spectrum to lower frequencies.
- 3. The shift on frequency spectrum observed in FST does not imply a change on the energy radiated.

Conclusion from preliminary study of validity of using once accelerometer:

Conclusion from preliminary study of validity of using once accelerometer:

- 1. Difference of 8dB were found between $\rm IL_a$ & $\rm IL_{ES}$. .
- Variations in the IL associated with the vibration acceleration are found to be small ⇒ location of the accelerometer is not of great importance.
- More parametric studies are required to accurately determine the validity of using one accelerometers for assessing a vibration reducing countermeasure.

Conclusion from re-radiated noise study:

Conclusion from re-radiated noise study:

- 1. Softer fastener shifts vibration spectra to low frequencies.
- 2. Softer fastener is a more efficient solution for vibration mitigation as it decrease vibration level of tunnel structures.
- 3. Noise levels in tunnels is influenced by rails noise radiation when vehicle is not considered.
- 4. Noise levels in the tunnel increases when rail vibration increases.

Publications

A methodology based on structural FEM-BEM and acoustic BEM models in 2.5D for the prediction of re-radiated noise in railway-induced ground-borne vibration problems

Journal of Vibration and Acoustics, 2019 (3) Dhananjay Ghangale, Aires Colaço, Pedro Alves Costa, Robert Arcos

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Journal of Computer Methods in Applied Mechanics and Engineering, (submitted) Dhananjay Ghangale, Robert Arcos, Arnau Clot, Julen Cayero, Jordi Romeu

Study of validity of using one accelerometer for assessing the efficiency of vibration reducing countermeasures

Journal of Tunnelling and Underground Space Technology, (in process) Dhananjay Ghangale, Robert Arcos, Arnau Clot, Jordi Romeu ction Novelties proposed in 2.5D FEM-BEM Accurate modelling of train response Vibration energy flow Re-radiated noise **Conclusions**OO OOOO OOOOO OOOOOO OOOOOOO

Future Work

Modelling the complete train/track/structure/soil/building system

- · Fast half-space solutions.
- Publication on track/soil coupling load.

Developing Fast Solvers

- MFS.
- SBM
- · General energy flow computation.

Hybrid Methodology

- For assessing the vibration reducing countermeasure where track already exist.
- Experimental measurements + numerical modelling.

¡Thank you for your attention!