# Solution of Matrix Eigenvalue Problem

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### Solution of Matrix Eigenvalue Problem

### Standard Matrix Eigenvalue Problem

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### Part I

Review of Previous Lecture

Jacobi Iteration Method

- Jacobi Iteration Method
- Gauss-Seidel Iteration Method

- Jacobi Iteration Method
- Gauss-Seidel Iteration Method
- Use of Software Packages

#### Part II

Solution of Matrix Eigenvalue Problem

# Standard Matrix Eigenvalue Problem: Introduction

Consider a system of equations in algebraic form

$$(a_{11} - \lambda)x_1 + a_{12}x_2 + a_{13}x_3 + \dots + a_{1n}x_n = 0$$

$$a_{21}x_1 + (a_{22} - \lambda)x_2 + a_{23}x_3 + \dots + a_{2n}x_n = 0$$

$$\vdots$$

$$a_{n1}x_1 + a_{n2}x_2 + a_{n3}x_3 + \dots + (a_{nn} - \lambda)x_n = 0$$

This is not a normal system of linear algebraic equations we're used to. For one, there are n equations, but n+1 unknowns (the  $x_i$  values, and also  $\lambda$ ). This particular system of equations is known as the standard eigenvalue problem.

### Form 1

The three forms shown are all algebraically equivalent. Any system of equations that can be expressed in these forms is a standard eigenvalue problem.

$$\begin{bmatrix} a_{11} - \lambda & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} - \lambda & \cdots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} - \lambda \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ \vdots \\ 0 \end{Bmatrix}$$

### Form 2

$$\begin{pmatrix}
\begin{bmatrix}
a_{11} & a_{12} & \cdots & a_{1n} \\
a_{21} & a_{22} & \cdots & a_{2n} \\
\vdots & \vdots & \ddots & \vdots \\
a_{n1} & a_{n2} & \cdots & a_{nn}
\end{bmatrix} - \lambda \begin{bmatrix}
1 & 0 & \cdots & 0 \\
0 & 1 & \cdots & 0 \\
\vdots & \vdots & \ddots & \vdots \\
0 & 0 & \cdots & 1
\end{bmatrix} \end{pmatrix} \begin{pmatrix}
x_1 \\
x_2 \\
\vdots \\
x_n
\end{pmatrix} = \begin{pmatrix}
0 \\
0 \\
\vdots \\
0
\end{pmatrix}$$

$$([A] - \lambda [I]) \{x\} = \{0\}$$

### Form 3

$$\begin{bmatrix} a_{11} & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} & \cdots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} \end{bmatrix} \begin{pmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{pmatrix} = \lambda \begin{pmatrix} x_1 \\ x_2 \\ \vdots \\ x_n \end{pmatrix}$$
$$[A] \{x\} = \lambda \{x\}$$

### Solvability of the Standard Eigenvalue Problem

Recall form 2 of the standard eigenvalue problem:

$$([A] - \lambda [I]) \{x\} = \{0\}$$

This system of equations has a solution for values of  $\lambda$  that cause the determinant of the coefficient matrix to equal 0, that is:

$$|[A] - \lambda [I]| = 0$$

### Characteristic Equation

Expanding out all the terms of the previous determinant

$$\begin{vmatrix} a_{11} - \lambda & a_{12} & \cdots & a_{1n} \\ a_{21} & a_{22} - \lambda & \cdots & a_{2n} \\ \vdots & \vdots & \ddots & \vdots \\ a_{n1} & a_{n2} & \cdots & a_{nn} - \lambda \end{vmatrix} = 0$$

yields a long polynomial in terms of  $\lambda$ . This polynomial will be nth order, and will therefore have n roots, each of which may be real or complex.

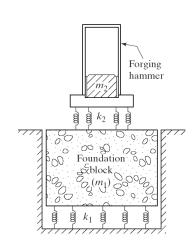
### General Eigenvalue Problem: Introduction

Many physical systems do not automatically present themselves as a standard eigenvalue problem, even though they can be reformatted as a standard eigenvalue problem. The form of a general eigenvalue problem is

$$[A] \{x\} = \lambda [B] \{x\}$$

where [A] and [B] are symmetric matrices of size  $n \times n$ .

A forging hammer of mass  $m_2$  is mounted on a concrete foundation block of mass  $m_1$ . The stiffnesses of the springs underneath the forging hammer and the foundation block are given by  $k_2$  and  $k_1$ , respectively.



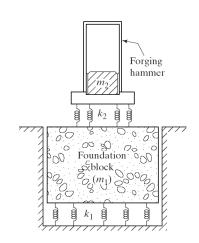
The system undergoes simple harmonic motion at one of its natural frequencies  $\omega$ . That is:

$$x_1(t) = \cos(\omega t + \phi_1)$$

$$x_2(t) = \cos(\omega t + \phi_2)$$

$$a_1(t) = -\omega^2 x_1(t)$$

$$a_2(t) = -\omega^2 x_2(t)$$

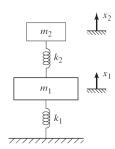


Each mass in the system obeys Newton's second law of motion, that is:

$$\Sigma F = ma$$

Forces on the foundation block:

- forces from the lower springs, which counteracts motion in the x direction at an amount  $-k_1x_1$
- forces from the upper springs, which act according to the amount of relative displacement of the masses  $m_1$  and  $m_2$ :  $-k_2(x_1-x_2)$



The equilibrium equation for the foundation mass is then

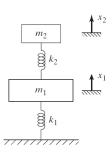
$$\Sigma F = ma$$

$$-k_1x_1 - k_2(x_1 - x_2) = m_1a$$

$$(-k_2 - k_1)x_1 + k_2x_2 = m_1a$$

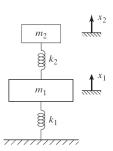
$$(-k_2 - k_1)x_1 + k_2x_2 = -m_1\omega^2x_1$$

$$(k_1 + k_2)x_1 - k_2x_2 = m_1\omega^2x_1$$



Similarly, the equilibrium equation for the forging hammer mass is

$$-k_2x_1 + k_2x_2 = m_2\omega^2x_2$$



So the two equations of motion are

$$(k_1 + k_2)x_1 - k_2x_2 = m_1\omega^2 x_1$$
$$-k_2x_1 + k_2x_2 = m_2\omega^2 x_2$$

or in matrix form

$$\begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \omega^2 \begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix}$$

This is a general eigenvalue problem

$$[A]\{x\} = \lambda[B]\{x\}$$

where [A] is the spring matrix,  $\{x\}$  is the vector of x values,  $\lambda$  is  $\omega^2$ , and [B] is the mass matrix.

### Eigenvalue Solutions in MATLAB: Standard Problems

The design of a mechanical component requires that the maximum principal stress to be less than the material strength. For a component subjected to arbitrary loads, the principal stresses  $\sigma$  are given by the solution of the equation

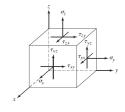
$$\begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{xy} & \sigma_{yy} & \tau_{yz} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{bmatrix} \begin{Bmatrix} l_x \\ l_y \\ l_z \end{Bmatrix} = \sigma \begin{Bmatrix} l_x \\ l_y \\ l_z \end{Bmatrix}$$

where the  $\sigma$  values represent normal stresses in the x, y, and z directions, and the  $\tau$  values represent shear stresses in the xy, xz, and yz planes. The I values represent direction cosines that define the principal planes on which the principal stress occurs.

## Eigenvalue Solutions in MATLAB: Standard Problems

Determine the principal stresses and principal planes in a machine component for the following stress condition

$$\begin{bmatrix} \sigma_{xx} & \tau_{xy} & \tau_{xz} \\ \tau_{xy} & \sigma_{yy} & \tau_{yz} \\ \tau_{xz} & \tau_{yz} & \sigma_{zz} \end{bmatrix} = \begin{bmatrix} 10 & 4 & -6 \\ 4 & -6 & 8 \\ -6 & 8 & 14 \end{bmatrix} MPa$$



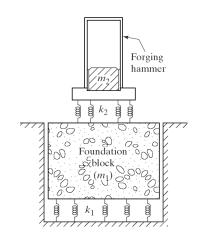
#### MATLAB Solution

### MATLAB Solution

### Eigenvalue Solutions in MATLAB: General Problems

Solve the forging hammer problem for the following values:

- $m_1 = 20000 \text{ kg}$
- $m_2 = 5000 \text{ kg}$
- $k_1 = 1 \times 10^7 \text{ N/m}$
- $k_2 = 5 \times 10^6 \text{ N/m}$



### Solving Eigenvalue Problems in MATLAB

Solving this eigenvalue problem will yield 2 eigenvalues equal to the square of the system's natural frequencies, and 2 corresponding x vector values that show the relative displacements of the  $m_1$  and  $m_2$  masses at those frequencies.

# MATLAB Solution (Part 1)

```
clear all;
% Define spring constants and masses
% for hammer and foundation block
k1 = 1e7;
k2 = 5e6;
m1 = 20000;
m2 = 5000;
% Define system stiffness matrix
K = \lceil k1 + k2 - k2 \rceil
      -k2 k2];
% Define system mass matrix
M = \lceil m \mid 1 \rangle
     0 m2];
```

# MATLAB Solution (Part 2)

```
% Solve general eigenvalue problem
[X,Omega2]=eig(K,M);
% diag(A) extracts the elements of the
% [A] matrix along the diagonal
Omega=diag(sqrt(Omega2));
% Scale column 1 of the [X] matrix by
% the row 1, column 1 X value
X(:,1)=X(:,1)/X(1,1);
% Scale column 2 of the [X] matrix by
% the row 1, column 2 X value
X(:,2)=X(:,2)/X(1,2);
Omega
χ
```

## MATLAB Solution (Results)

```
>> rao_ex42

Omega =

    18.9634

    37.2879

X =

    1.0000    1.0000

    1.5616    -2.5616
```

### Homework

Continue working on the Gauss-Seidel and other homework problems already assigned.