(Ch.3) Dynamics Review - Modeling Mechanical Systems

Translational Motion

Position: x(t) Velocity: $y(t) = \frac{dx}{dt} = \dot{x}(t)$ Acceleration: $a(t) = \frac{dy}{dt} = \frac{d^2x}{dt^2} = \ddot{x}(t)$

Rotational Motion

Angular Position: $\theta(t)$ Angular Velocity: $\omega(t) = \frac{d\theta}{dt} = \dot{\theta}(t)$

Angular Acceleration: $\alpha(t) = \frac{d\omega}{dt} = \frac{d^2\theta}{dt^2} = \ddot{\theta}(t)$

Determining Equation of Motion

1. Draw FBD (5)

2. Apply Newton's 2nd Law or use an Energy Method

Newton's 2nd Law Approach

Translational: EF=ma m= mass

Rotational: $\angle M_0 = I_0 \, \alpha$ $= I_0 \, \alpha$

Energy Method

Kinetic Energy: Translational: $T = \frac{1}{2}mv^2$ Rotational: $T = \frac{1}{2}I_0\omega^2$

Potential Energy: Gravity: V = mgh (g = accel. of gravity, h = vertical displacement)

Translational Spring: $V = \frac{1}{2}KX^2$ K = linear spring stiffnessRotational Spring: $V = \frac{1}{2}K_E \theta^2$ $K_E = torsional spring stiffness$

If only conservative forces are applied (i.e. no damping, friction, etc.), then

T+V = constant (conservation of energy)

 $\Delta T_{+} \Delta V = 0$ or $T_{z+} V_{z} = T_{i} + V_{i}$

 $\Delta T = T_z - T_1 = \frac{1}{2} m(V_2^2 - V_1^2)$

 $\Delta V = V_2 - V_1 = \frac{1}{2} K (X_2^2 - X_1^2) - \text{for a spring}$

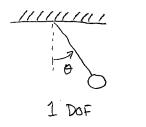
EOM can be found by realizing that

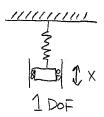
 $\frac{d}{dt}(T+V) = \frac{d}{dt}(const) = 0$

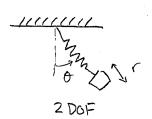
 $T_2 + V_2 = T_1 + V_1 + W_{1\rightarrow 2}$ W_{1 \rightarrow 2} is work of nonconservative forces.

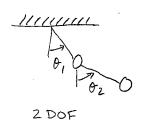
This leads to a more complex method called the Lagrange Method. We won't cover in this rourse. Degrees of Freedom

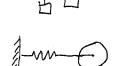
DOF is the number of coordinates required to characterize a system











(translation + rotation

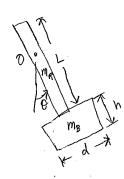
Need one equation of motion per DOF to describe system

Example (Ex 3.1.2)

Find EOM for following systems

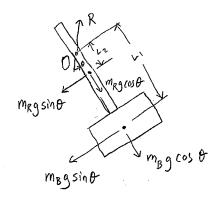
(a) FBD:

$$uN \rightarrow m \leftarrow f$$
 $\xi F_x = m\dot{x} = f - \mu mg \Rightarrow m\ddot{x} + \mu mg = f$



Pendulum Both rod + block have mass, MR, MB L, , Lz => distances from centers of mass to pivot point, O.

FBD:



inertia about symmetry axis

Newton's Method

$$\xi M_0 = I_0 \ddot{\theta} = (I_{R0} + I_{B0}) \ddot{\theta}$$

Parallel Axis Theorem: I= Is+md2 (p. 123)

Rod:
$$I_s = \frac{1}{12} m_R L^2 \Rightarrow I_{Ro} = \frac{1}{12} m_R L^2 + m_R L_2$$

Block: $I_s = \frac{1}{12} m_B (h^2 + d^2) \Rightarrow I_{Bo} = \frac{1}{12} m_B (h^2 + d^2) + m_R L_1$

$$(I_{Ro} + I_{Bo})\ddot{\theta} = -m_{R}g\sin\theta L_{z} - m_{B}g\sin\theta L_{t}$$

 $(I_{Ro} + I_{Bo})\ddot{\theta} + (m_{R}gL_{z} + m_{B}gL_{t})\sin\theta = 0$

Energy Method

$$T + V = cst$$

Recall:
$$T = \frac{1}{2}I_0\omega^2 = \frac{1}{2}I_0\dot{\theta}^2$$
 Potential Energy: Losson h=

* Differentiate w.r.t. time: use chain rule =>
$$\frac{d}{dt}\dot{\theta}^2 = 2\dot{\theta}\dot{\theta}$$
, $\frac{d}{dt}\cos\theta = -\sin\theta\dot{\theta}$

$$Z/Z\left(I_{Ro} + I_{Bo}\right)\dot{\theta}\ddot{\theta} + m_R gL_Z\sin\theta\dot{\theta} + m_B gL_Z\sin\theta\dot{\theta} = 0$$