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updated by D.R. van der Heul

# ELEMENTS OF COMPUTATIONAL FLUID DYNAMICS

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#### **Preface**

The technological value of computational fluid dynamics has become undisputed. A capability has been established to compute flows that can be investigated experimentally only at reduced Reynolds numbers, or at greater cost, or not at all, such as the flow around a space vehicle at re-entry, or a loss-of-coolant accident in a nuclear reactor. Large commercial computational fluid dynamics computer codes have arisen, and found widespread use in industry. Users of these codes need to be familiar with the basic principles. It has been observed on numerous occasions, that even simple flows are not correctly predicted by advanced computational fluid dynamics codes, if used without sufficient insight in both the numerics and the physics involved. This course aims to elucidate some basic principles of computational fluid dynamics.

Because the subject is vast we have to confine ourselves here to just a few aspects. A more complete introduction is given in Wesseling (2001), and other sources quoted there. Occasionally, we will refer to the literature for further information. But the student will be examined only about material presented in these lecture notes.

Fluid dynamics is governed by partial differential equations. These may be solved numerically by finite difference, finite volume, finite element and spectral methods. In engineering applications, finite difference and finite volume methods are predominant. We will confine ourselves here to finite difference and finite volume methods.

Although most practical flows are turbulent, we restrict ourselves here to laminar flow, because this book is on numerics only. The numerical principles uncovered for the laminar case carry over to the turbulent case. Furthermore, we will discuss only incompressible flow. Considerable attention is given to the convection-diffusion equation, because much can be learned from this simple model about numerical aspects of the Navier-Stokes equations. One chapter is devoted to direct and iterative solution methods.

Errata and MATLAB software related to a number of examples discussed in these course notes may be obtained via the author's website, to be found at ta.twi.tudelft.nl/nw/users/wesseling (see under "Information for students' / "College WI4 011 Numerieke Stro-

mingsleer")

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P. Wesseling

### Preface to the updated edition

The content of the course 'Numerieke Stromingsleer" have been adapted to eliminate the overlap with concurring courses on computational fluid dynamics and with the content of the course 'WI 4201 Scientific Computing'. In particular the subject of the earlier Chapter 6, *iterative solution methods*, is extensively discussed in the latter course. To be able to understand the fundamentals of structured grid generation same basic tensor analysis is discussed. However, students that want to pursue in the field of computational fluid dynamics are advised to take the course 'WI 4037 Tensor Analysis'

The course is (still) meant as an in depth mathematical discussion of the mere basics of the field of computational fluid dynamics, as opposed to other courses that will offer a more shallow discussion of many more aspects of the field.

The original lecture notes have been augmented with additional remarks and in some cases additional steps in the derivations have been added to assist the reader in understanding the concepts, without being discouraged by the mathematical analysis, which is straightforward but is regarded by some as tedious.

The original content is extended with a discussion of grid generation techniques. It is important to understand, that apart from analysis of the computed results, the main workload for the computational fluid dynamics practitioner is in the generation of the grid. Therefore, there is a continuing interest in industry to reduce the workload (=cost) in grid generation by automating the process. This can be achieved for both 'structured' and 'unstructured' computational grids, using advanced numerical techniques. Again, an indepth discussion of the basics is given, as opposed to briefly touching on the many different flavours that exist.

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D.R. van der Heul

### Contents

Pre	eface		J
Pre	eface	to the updated edition	III
1.	The 1.1 1.2 1.3 1.4	basic equations of fluid dynamics  Introduction  Vector analysis  The total derivative and the transport theorem  Conservation of mass	1 1 1 4 5
	1.5 1.6 1.7	Conservation of mass  Conservation of momentum  The convection-diffusion equation  Summary of this chapter	11 13
2.	The 2.1 2.2 2.3	stationary convection-diffusion equation in one dimension Introduction	15 15 16 20
3.		stationary convection-diffusion equation in two dimensis	41 41 42 50
4.	The 4.1 4.2 4.3 4.4 4.5	nonstationary convection-diffusion equation Introduction A numerical example Convergence, consistency and stability Fourier stability analysis Numerical experiments	59 59 60 62 66 77
5.	The 5.1 5.2 5.3 5.4 5.5 5.6	incompressible Navier-Stokes equations Introduction  Equations of motion and boundary conditions  Spatial discretization on staggered grid  Spatial discretization on colocated grid  Temporal discretization on staggered grid  Numerical experiments	81 81 82 84 89 94

6.	Disc	cretisa	tion on an unstructured grid	107		
	6.1	1 Introduction				
	6.2	Discretisation of the incompressible Navier-Stokes equations on				
		an uns	structured grid	107		
		6.2.1	Structure in an unstructured grid	108		
		6.2.2	The rotational form of the incompressible Navier-Stokes			
			equations	108		
		6.2.3	The divergence form of the Navier-Stokes equations	111		
7.	Disc	cretisa	tion on a boundary conforming grid	113		
	7.1		asic idea	113		
	7.2	Some	tensor calculus	113		
	7.3	The co	oordinate mapping	114		
		7.3.1	Definition of the basis vectors	114		
		7.3.2	Construction of the basis vectors	114		
		7.3.3	Covariant and contravariant vectors	116		
		7.3.4	Representation of $u$	117		
		7.3.5	The permutation symbol $\varepsilon_{\alpha\beta\gamma}$	118		
		7.3.6	The metric tensor	119		
		7.3.7	Contraction	120		
		7.3.8	The inner product	120		
		7.3.9	Physical component	120		
		7.3.10	Christoffel symbols	121		
			Differentiating the metric tensor	123		
			Differentiating a vector field	124		
			Formulating equations in coordinate invariant form	125		
			Divergence operator	125		
			Laplace operator	126		
			An example computation			
			Navier-Stokes equations			
8.	Intr	oducti	ion to classical grid generation	129		
	8.1		ng point	129		
	8.2		ured grid generation	130		
		8.2.1	Algebraic grid generation techniques			
	8.3		ential models for grid generation	134		
	0.0	8.3.1	The Winslow equations			
		8.3.2	The Thompson-Thames-Mastin (TTM) equations			
		8.3.3	Source function for control of grid density			
		8.3.4	Source function for control of direction of the grid lines .			
		0.0.1	service for control of direction of the Stiff lines .	-01		
Rof	foron	COS		1/19		

#### 1. The basic equations of fluid dynamics

#### 1.1 Introduction

Fluid dynamics is a classic discipline. The physical principles governing the flow of simple fluids and gases, such as water and air, have been understood since the times of Newton. Since about 1950 classic fluid dynamics finds itself in the company of computational fluid dynamics. This newer discipline still lacks the elegance and unification of its classic counterpart, and is in a state of rapid development.

Good starting points for exploration of the Internet for material related to computational fluid dynamics is the following website:

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www.cfd-online.com/
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The site contains a lot of information on available textbooks and discussion boards related to different topics and packages to post your questions and share your ideas.

Readers well-versed in theoretical fluid dynamics may skip the remainder of this chapter, perhaps after taking note of the notation introduced in the next section. But those less familiar with this discipline will find it useful to continue with the present chapter.

The purpose of this chapter is:

- To introduce some notation that will be useful later;
- To recall some basic facts of vector analysis;
- To introduce the governing equations of laminar incompressible fluid dynamics:
- To explain that the Reynolds number is usually very large. In later chapters this will be seen to have a large impact on numerical methods.

#### 1.2 Vector analysis

#### Cartesian tensor notation

We assume a right-handed Cartesian coordinate system  $(x_1, x_2, ..., x_d)$  with d the number of space dimensions. Bold-faced lower case Latin letters denote vectors, for example,  $\mathbf{x} = (x_1, x_2, ..., x_d)$ . Greek letters denote scalars. In *Cartesian tensor notation*, which we shall often use, differentiation is denoted as follows:

$$\phi_{,\alpha} = \frac{\partial \phi}{\partial x_{\alpha}}.\tag{1.1}$$

Greek subscripts refer to coordinate directions, and the *summation convention* is used: summation takes place over Greek indices that occur twice in a term or product. The reason for our preference for this compact notation will become clear in the discussion of formulation in *general* coordinates. In the latter case the 'standard' notation becomes totally impractical.

#### Examples

Inner product:  $\mathbf{u} \cdot \mathbf{v} = u_{\alpha} v_{\alpha} = \sum_{\alpha=1}^{d} u_{\alpha} v_{\alpha}$ 

Laplace operator:  $\nabla^2 \phi = \phi_{,\alpha\alpha} = \sum_{\alpha=1}^d \frac{\partial^2 \phi}{\partial x_{\alpha}^2}$ 

Note that 
$$u_{\alpha} + v_{\alpha}$$
 does not mean  $\sum_{\alpha=1}^{d} (u_{\alpha} + v_{\alpha})$  (why?)

We will also use *vector notation*, instead of the *subscript notation* just explained, and may write  $\nabla \cdot \mathbf{u}$ , if this is more elegant or convenient than the tensor equivalent  $u_{\alpha,\alpha}$ ; and sometimes we write grad  $\phi$  or  $\nabla \phi$  for the vector  $(\phi_{,1}, \phi_{,2}, \phi_{,3})$ .

The Kronecker delta  $\delta_{\alpha\beta}$  is defined by:

$$\delta_{11} = \delta_{22} = \dots = \delta_{dd} = 1, \quad \delta_{\alpha\beta} = 0, \ \alpha \neq \beta$$

where d is the number of space dimensions.

#### Divergence theorem

We will need the following fundamental theorem:

**Theorem 1.2.1.** For any volume  $V \subset \mathbb{R}^d$  bounded by a piecewise smooth closed surface S and any differentiable scalar field  $\phi$  we have

$$\int\limits_{V} \phi_{,\alpha} dV = \int\limits_{S} \phi n_{\alpha} dS \; ,$$

where  $\mathbf{n}$  is the outward unit normal on S (and  $n_{\alpha}$  the component of  $\mathbf{n}$  in the  $\alpha$  direction.

For a proof, see for example Aris (1962). The proof is given for a *simple solid* region (a cylindrical region with the cylinder axis aligned with one of the coordinate directions) and is generalized to a general region by merging simple solid regions.

A direct consequence of this theorem is:

**Theorem 1.2.2.** (Divergence theorem).

For any volume  $V \subset \mathbb{R}^d$  with piecewise smooth closed surface S and any differentiable vector field  $\mathbf{u}$  we have

$$\int\limits_{V}u_{\alpha,\alpha}dV=\int\limits_{S}n_{\alpha}u_{\alpha}dS\;,$$

where n is the outward unit normal on S.

*Proof.* Apply Theorem 1.2.1 with  $\phi_{,\alpha}=u_{\alpha},\ \alpha=1,2,...,d$  successively and add.  $\Box$ 

A vector field satisfying  $u_{\alpha,\alpha} = 0$  is called *solenoidal*.

#### The streamfunction

In two dimensions, if for a given velocity field  ${\bf u}$  there exists a function  $\psi$  such that

$$\psi_{,1} = -u_2, \quad \psi_{,2} = u_1,$$

then such a function is called the *streamfunction*. For the streamfunction to exist it is obviously necessary that  $\psi_{,12} = \psi_{,21}$ ; therefore we must have  $u_{1,1} = -u_{2,2}$ , or div  $\mathbf{u} = 0$ . Hence, two-dimensional solenoidal vector fields have a streamfunction. The normal to an isoline  $\psi(\mathbf{x}) = \text{constant}$  is parallel to  $\nabla \psi = (\psi_{,1}, \psi_{,2})$ ; therefore the vector  $\mathbf{u} = (\psi_{,2}, -\psi_{,1})$  is tangential to this isoline. Streamlines are curves that are everywhere tangential to  $\mathbf{u}$ . We see that in two dimensions the streamfunction is constant along streamlines. Later, this fact will provide us with a convenient way to compute streamline patterns numerically.

#### Potential flow

The *curl* of a vector field is defined by

$$\nabla \times \mathbf{u} = \begin{pmatrix} u_{3,2} - u_{2,3} \\ u_{1,3} - u_{3,1} \\ u_{2,1} - u_{1,2} \end{pmatrix} .$$

That is, the  $x_1$ -component of the vector curl**u** is  $u_{3,2} - u_{2,3}$ , etc. Often, the curl is called rotation, and a vector field satisfying  $\nabla \times \mathbf{u} = \mathbf{0}$  is called *irrotational*. In two dimensions, the curl is obtained by putting the third component and  $\partial/\partial x_3$  equal to zero. This gives

$$\nabla \times \mathbf{u} = u_{2,1} - u_{1,2}$$
.

It can be shown (cf. Aris (1962)) that if a vector field  $\mathbf{u}$  satisfies  $\nabla \times \mathbf{u} = \mathbf{0}$  there exists a scalar field  $\varphi$  such that

$$\mathbf{u} = \nabla \varphi, \quad \text{or} \quad u_{\alpha} = \varphi_{,\alpha}$$
 (1.2)

#### 1. The basic equations of fluid dynamics

(or  $u_{\alpha} = \varphi_{,\alpha}$ ). The scalar  $\varphi$  is called the *potential*, and flows with velocity field  $\mathbf{u}$  satisfying (1.2) are called potential flows or *irrotational* flows (since  $\nabla \times \nabla \varphi = 0$ , cf. Exercise 1.2.3). The potential flow model assumes the flow to be inviscid and solenoidal. Because of the simplicity of the model numerical solutions can be obtained very quickly. In those cases where viscous effects are not important, the potential flow model can be relatively accurate, e.g. in the computation of the lift of thin airfoils at small angle of attack.

**Exercise 1.2.1.** Prove Theorem 1.2.1 for the special case that V is the unit cube.

**Exercise 1.2.2.** Show that  $\nabla \times \mathbf{u}$  is solenoidal.

**Exercise 1.2.3.** Show that  $\operatorname{curl} \operatorname{grad} \varphi = 0$ .

**Exercise 1.2.4.** Show that  $\delta_{\alpha\alpha} = d$ .

#### 1.3 The total derivative and the transport theorem

#### **Streamlines**

4

We repeat: a *streamline* is a curve that is everywhere tangent to the velocity vector  $\mathbf{u}(t, \mathbf{x})$  at a given time t. Hence, a streamline may be parameterized with a parameter s such that a streamline is a curve  $\mathbf{x} = \mathbf{x}(s)$  defined by

$$d\mathbf{x}/ds = \mathbf{u}(t, \mathbf{x})$$
.

#### The total derivative

Let  $\mathbf{x}(t, \mathbf{y})$  be the position of a material particle at time t > 0, that at time t = 0 had initial position  $\mathbf{y}$ . Obviously, the velocity field  $\mathbf{u}(t, \mathbf{x})$  of the flow satisfies

$$\mathbf{u}(t, \mathbf{x}) = \frac{\partial \mathbf{x}(t, \mathbf{y})}{\partial t} \ . \tag{1.3}$$

The time-derivative of a property  $\phi$  of a material particle, called a material property (for example its temperature), is denoted by  $D\phi/Dt$ . This is called the total derivative. All material particles have some  $\phi$ , so  $\phi$  is defined everywhere in the flow, and is a scalar field  $\phi(t, \mathbf{x})$ . We have

$$\frac{D\phi}{Dt} \equiv \frac{\partial}{\partial t}\phi[t, \mathbf{x}(t, \mathbf{y})] , \qquad (1.4)$$

where the partial derivative has to be taken with y constant, since the total derivative tracks variation for a particular material particle. We obtain

$$\frac{D\phi}{Dt} = \frac{\partial\phi}{\partial t} + \phi_{,\alpha} \frac{\partial x_{\alpha}(t, \mathbf{y})}{\partial t} \ .$$

By using (1.3) we get

$$\frac{D\phi}{Dt} = \frac{\partial\phi}{\partial t} + u_{\alpha}\phi_{,\alpha} \ .$$

#### The transport theorem

A material volume V(t) is a volume of fluid that moves with the flow and consists permanently of the same material particles.

**Theorem 1.3.1.** (Reynolds's transport theorem)

For any material volume V(t) and differentiable scalar field  $\phi$  we have

$$\frac{d}{dt} \int_{V(t)} \phi dV = \int_{V(t)} \left( \frac{\partial \phi}{\partial t} + (\phi u_{\alpha})_{,\alpha} \right) dV . \tag{1.5}$$

Not the fact that the integration volume is **time-dependent**. For a proof, see Sect. 1.3 of Wesseling (2001).

We are now ready to formulate the governing equations of fluid dynamics, which consist of the conservation laws for mass, momentum and energy.

#### 1.4 Conservation of mass

#### Continuum hypothesis

The dynamics of fluids is governed by the conservation laws of classical physics, namely conservation of mass, momentum and energy. From these laws partial differential equations are derived and, under appropriate circumstances, simplified. It is customary to formulate the conservation laws under the assumption that the fluid is a continuous medium (continuum hypothesis). Physical properties of the flow, such as density and velocity can then be described as time-dependent scalar or vector fields on  $\mathbb{R}^2$  or  $\mathbb{R}^3$ , for example  $\rho(t, \mathbf{x})$  and  $\mathbf{u}(t, \mathbf{x})$ .

#### The mass conservation equation

The mass conservation law says that the rate of change of mass in an arbitrary material volume V(t) equals the rate of mass production in V(t). This can be expressed as

$$\frac{d}{dt} \int_{V(t)} \rho dV = \int_{V(t)} \sigma dV , \qquad (1.6)$$

where  $\rho(t, \mathbf{x})$  is the density of the material particle at time t and position  $\mathbf{x}$ , and  $\sigma(t, \mathbf{x})$  is the rate of mass production per volume. In practice,  $\sigma \neq 0$  only in multiphase flows, in which case (1.6) holds for each phase separately. We take  $\sigma = 0$ , and use the transport theorem to obtain

$$\int\limits_{V(t)} \left( \frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) \right) dV = 0 \; .$$

Since this holds for every V(t) the integrand must be zero:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0. \tag{1.7}$$

This is the mass conservation law, also called the continuity equation.

#### Incompressible flow

An incompressible flow is a flow in which the density of each material particle remains the same during the motion:

$$\rho[t, \mathbf{x}(t, \mathbf{y})] = \rho(0, \mathbf{y}). \tag{1.8}$$

Hence

$$\frac{D\rho}{Dt} = 0. ag{1.9}$$

Because

$$\nabla \cdot (\rho \mathbf{u}) = \rho \nabla \cdot \mathbf{u} + \mathbf{u} \cdot \nabla \rho ,$$

it follows from the mass conservation law (1.7) that

$$\nabla \cdot \mathbf{u} = 0 \ . \tag{1.10}$$

This is the form that the mass conservation law takes for incompressible flow.

Sometimes incompressibility is erroneously taken to be a property of the fluid rather than of the flow. But it may be shown that compressibility depends only on the speed of the flow, see Sect. 1.12 of Wesseling (2001). If the magnitude of the velocity of the flow is of the order of the speed of sound in the fluid ( $\sim 340$  m/s in air at sea level at 15°C,  $\sim 1.4$  km/s in water at 15°C, depending on the amount of dissolved air) the flow is compressible; if the velocity is much smaller than the speed of sound, incompressibility is a good approximation. In liquids, flow velocities anywhere near the speed of sound cannot normally be reached, due to the enormous pressures involved and the phenomenon of cavitation.

#### 1.5 Conservation of momentum

#### Body forces and surface forces

Newton's law of conservation of momentum implies that the rate of change of momentum of a material volume equals the total force on the volume. There are body forces and surface forces. A body force acts on a material particle, and is proportional to its mass. Let the volume of the material particle be dV(t) and let its density be  $\rho$ . Then we can write

body force = 
$$\mathbf{f}^b \rho dV(t)$$
. (1.11)

A surface force works on the surface of V(t) and is proportional to area. The surface force working on a surface element dS(t) of V(t) can be written as

surface force = 
$$\mathbf{f}^s dS(t)$$
. (1.12)

#### Conservation of momentum

The law of conservation of momentum applied to a material volume gives

$$\frac{d}{dt} \int_{V(t)} \rho u_{\alpha} dV = \int_{V(t)} \rho f_{\alpha}^{b} dV + \int_{S(t)} f_{\alpha}^{s} dS. \qquad (1.13)$$

By substituting  $\phi = \rho u_{\alpha}$  in the transport theorem (1.5), this can be written as

$$\int_{V(t)} \left[ \frac{\partial \rho u_{\alpha}}{\partial t} + (\rho u_{\alpha} u_{\beta})_{,\beta} \right] dV = \int_{V(t)} \rho f_{\alpha}^{b} dV + \int_{S(t)} f_{\alpha}^{s} dS . \tag{1.14}$$

It may be shown (see Aris (1962)) there exist nine quantities  $\tau_{\alpha\beta}$  such that

$$f_{\alpha}^{s} = \tau_{\alpha\beta} n_{\beta} , \qquad (1.15)$$

where  $\tau_{\alpha\beta}$  is the *stress tensor* and **n** is the outward unit normal on dS. By applying Theorem 1.2.1 with  $\phi$  replaced by  $\tau_{\alpha\beta}$  and  $n_{\alpha}$  by  $n_{\beta}$ , equation (1.14) can be rewritten as

$$\int_{V(t)} \left[ \frac{\partial \rho u_{\alpha}}{\partial t} + (\rho u_{\alpha} u_{\beta})_{,\beta} \right] dV = \int_{V(t)} (\rho f_{\alpha}^b + \tau_{\alpha\beta,\beta}) dV .$$

Since this holds for every V(t), we must have

$$\frac{\partial \rho u_{\alpha}}{\partial t} + (\rho u_{\alpha} u_{\beta})_{,\beta} = \tau_{\alpha\beta,\beta} + \rho f_{\alpha}^{b} , \qquad (1.16)$$

which is the momentum conservation law. The left-hand side is called the *inertia* term, because it comes from the inertia of the mass of fluid contained in V(t) in equation (1.13).

An example where  $\mathbf{f}^b \neq 0$  is stratified flow under the influence of gravity.

#### Constitutive relation

In order to complete the system of equations it is necessary to relate the stress tensor to the local motion of the fluid. Such a relation is called a *constitutive relation*. A full discussion of constitutive relations would lead us too far. The simplest constitutive relation is (see Batchelor (1967))

$$\tau_{\alpha\beta} = -p\delta_{\alpha\beta} + 2\mu(e_{\alpha\beta} - \frac{1}{3}\Delta\delta_{\alpha\beta}), \qquad (1.17)$$

where p is the pressure,  $\delta_{\alpha\beta}$  is the Kronecker delta,  $\mu$  is the dynamic viscosity,  $e_{\alpha\beta}$  is the rate of strain tensor, defined by

$$e_{\alpha\beta} = \frac{1}{2}(u_{\alpha,\beta} + u_{\beta,\alpha}) ,$$

and

$$\Delta = e_{\alpha\alpha} = u_{\alpha,\alpha} .$$

The quantity  $\nu = \mu/\rho$  is called the *kinematic viscosity*.. This relation is based on:

- The experimentally confirmed assumption that there is a *linear* relation between the stress tensor and gradient of the velocity.
- The fact that the stress tensor should be symmetric
- The fact that the deviatoric stress tensor by definition does not contribute to the mean normal stress, which should be equal to the pressure.

The relation (1.17) was derived by Saint-Venant (1842!) and Stokes (1845). In many fluids and gases  $\mu$  depends on temperature, but not on pressure. Fluids satisfying (1.17) are called *Newtonian fluids*. Examples are gases and liquids such as water and mercury. Examples of non-Newtonian fluids are polymers and blood.

#### The Navier-Stokes equations

Substitution of (1.17) in (1.16) gives

$$\frac{\partial \rho u_{\alpha}}{\partial t} + (\rho u_{\alpha} u_{\beta})_{,\beta} = -p_{,\alpha} + 2[\mu(e_{\alpha\beta} - \frac{1}{3}\Delta\delta_{\alpha\beta})]_{,\beta} + \rho f_{\alpha}^{b}. \tag{1.18}$$

These are the Navier-Stokes equations. The terms in the left-hand side are due to the inertia of the fluid particles, and are called the *inertia terms*. The first term on the right represents the pressure force that works on the fluid particles, and is called the *pressure term*. The second term on the right represents the friction force, and is called the *viscous term*. The third term on the right is the *body force*.

Because of the continuity equation (1.7), one may also write

$$\rho \frac{Du_{\alpha}}{Dt} = -p_{,\alpha} + 2[\mu(e_{\alpha\beta} - \frac{1}{3}\Delta\delta_{\alpha\beta})]_{,\beta} + \rho f_{\alpha}^{b}.$$
 (1.19)

In incompressible flows  $\Delta = 0$ , and we get

$$\rho \frac{Du_{\alpha}}{Dt} = -p_{,\alpha} + 2(\mu e_{\alpha\beta})_{,\beta} + \rho f_{\alpha}^{b} . \tag{1.20}$$

These are the incompressible Navier-Stokes equations. If, furthermore,  $\mu = \text{constant}$  then we can use  $u_{\beta,\alpha\beta} = (u_{\beta,\beta})_{\alpha} = 0$  to obtain

$$\rho \frac{Du_{\alpha}}{Dt} = -p_{,\alpha} + \mu u_{\alpha,\beta\beta} + \rho f_{\alpha}^{b} . \tag{1.21}$$

This equation was first derived by Navier (1823), Poisson (1831), de Saint-Venant (1843) and Stokes (1845). Its vector form is

$$\rho \frac{D\mathbf{u}}{Dt} = -\nabla p + \mu \nabla^2 \mathbf{u} + \rho \mathbf{f}^b ,$$

where  $\nabla^2$  is the Laplace operator. The quantity

$$\frac{D\mathbf{u}}{Dt} = \frac{\partial \mathbf{u}}{\partial t} + u_{\alpha}\mathbf{u}_{,\alpha}$$

is sometimes written as

$$\frac{D\mathbf{u}}{Dt} = \frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} .$$

#### Making the equations dimensionless

In fluid dynamics there are exactly four independent physical units: those of length, velocity, mass and temperature, to be denoted by L, U, M and  $T_r$ , respectively. From these all other units can be and should be derived in order to avoid the introduction of superfluous coefficients in the equations. For instance, the appropriate unit of time is L/U; the unit of force F follows from Newton's law as  $MU^2/L$ . Often it is useful not to choose these units arbitrarily, but to derive them from the problem at hand, and to make the equations dimensionless. This leads to the identification of the dimensionless parameters that govern a flow problem. An example follows.

#### The Reynolds number

Let L and U be typical length and velocity scales for a given flow problem, and take these as units of length and velocity. The unit of mass is chosen as  $M = \rho_r L^3$  with  $\rho_r$  a suitable value for the density, for example the density in the flow at upstream infinity, or the density of the fluid at rest. Dimensionless variables are denoted by a prime:

$$\mathbf{x}' = \mathbf{x}/L, \quad \mathbf{u}' = \mathbf{u}/U, \quad \rho' = \rho/\rho_r.$$
 (1.22)

In dimensionless variables, equation (1.18) takes the following form:

$$\frac{L}{U} \frac{\partial \rho' u'_{\alpha}}{\partial t} + (\rho' u'_{\alpha} u'_{\beta})_{,\beta} = -\frac{1}{\rho_r U^2} p_{,\alpha} + \frac{2}{\rho_r U L} \{ \mu (e'_{\alpha\beta} - \frac{1}{3} \Delta' \delta_{\alpha\beta}) \}_{,\beta} + \frac{L}{U^2} \rho' f_{\alpha}^b ,$$
(1.23)

where now the subscript ,  $\alpha$  stands for  $\partial/\partial x'_{\alpha}$ , and  $e'_{\alpha\beta} = \frac{1}{2}(u'_{\alpha,\beta} + u'_{\beta,\alpha})$ ,  $\Delta' = e'_{\alpha\alpha}$ . We introduce further dimensionless quantities as follows:

$$t' = Ut/L, \quad p' = p/\rho_r U^2, \quad (\mathbf{f}^b)' = \frac{L}{U^2} \mathbf{f}^b.$$
 (1.24)

By substitution in (1.23) we obtain the following *dimensionless form* of the Navier-Stokes equations, deleting the primes:

$$\frac{\partial \rho u_{\alpha}}{\partial t} + (\rho u_{\alpha} u_{\beta})_{,\beta} = -p_{,\alpha} + 2\{\operatorname{Re}^{-1}(e_{\alpha\beta} - \frac{1}{3}\Delta\delta_{\alpha\beta})\}_{,\beta} + f_{\alpha}^{b},$$

where the Reynolds number Re is defined by

$$Re = \frac{\rho_r UL}{\mu} .$$

The dimensionless form of (1.21) is, if  $\rho = \text{constant} = \rho_r$ ,

$$\frac{Du_{\alpha}}{Dt} = -p_{,\alpha} + \operatorname{Re}^{-1}u_{\alpha,\beta\beta} + f_{\alpha}^{b} . \tag{1.25}$$

The transformation (1.22) shows that the inertia term is of order  $\rho_r U^2/L$  and the viscous term is of order  $\mu U/L^2$ . Hence, Re is a measure of the ratio of inertial and viscous forces in the flow. This can also be seen immediately from

equation (1.25). For Re  $\gg 1$  inertia dominates, for Re  $\ll 1$  friction (the viscous term) dominates. Both are balanced by the pressure gradient.

In the case of constant density, equations (1.25) and (1.10) form a complete system of four equations with four unknowns. The solution depends on the single dimensionless parameter Re only. What values does Re have in nature? At a temperature of 15°C and atmospheric pressure, for air we have for the kinematic viscosity  $\mu/\rho = 1.5 * 10^{-5} \text{ m}^2/\text{s}$ , whereas for water  $\mu/\rho = 1.1 * 10^{-6}$  m<sup>2</sup>/s. In the International Civil Aviation Organization Standard Atmosphere,  $\mu/\rho = 4.9 * 10^{-5} \text{ m}^2/\text{s}$  at an altitude of 12.5 km. This gives for the flow over an aircraft wing in cruise condition at 12.5 km altitude with wing cord L=3 m and U=900 km/h: Re =  $1.5*10^7$ . In a windtunnel experiment at sea-level with L = 0.5 m and U = 25 m/s we obtain Re =  $8.3 * 10^5$ . For landing aircraft at sea-level with  $L=3~\mathrm{m}$  and  $U=220~\mathrm{km/h}$  we obtain Re =  $1.2 * 10^7$ . For a house in a light wind with L = 10 m and U = 0.5 m/s we have Re =  $3.3 * 10^5$ . Air circulation in a room with L = 4 m and U = 0.1 m/s gives Re =  $2.7 * 10^4$ . A large ship with L = 200 m and U = 7 m/s gives  $Re = 1.3*10^8$ , whereas a yacht with L = 7 m and U = 3 m/s has  $Re = 1.9*10^7$ . A small fish with L = 0.1 m and U = 0.2 m/s has  $Re = 1.8 * 10^4$ .

All these very different examples have in common that  $Re \gg 1$ , which is indeed almost the rule in flows of industrial and environmental interest. One might think that flows around a given shape will be quite similar for different values of Re, as long as Re  $\gg 1$ , but nothing is farther from the truth. At Re =  $10^7$  a flow may be significantly different from the flow at  $Re = 10^5$ , in the same geometry. This strong dependence on Re complicates predictions based on scaled down experiments. Modern cryogenic wind tunnels allow testing of scaled models at the correct Mach number and Reynolds number, by cooling the wind tunnel medium to cryogenic temperatures. Performing wind tunnel tests is expensive, whereas cryogenic wind tunnel testing is really, really expensive. Therefore computational fluid dynamics plays an important role in extrapolation to full scale. The rich variety of solutions of (1.25) that evolves as Re  $\rightarrow \infty$  is one of the most surprising and interesting features of fluid dynamics, with important consequences for technological applications. A 'route to chaos' develops as  $\text{Re} \to \infty$ , resulting in turbulence. Intricate and intriguing flow patterns occur, accurately rendered in masterful drawings by Leonardo da Vinci, and photographically recorded in Hinze (1975), Nakayama and Woods (1988), Van Dyke (1982) and Hirsch (1988).

Turbulent flows are characterized by small rapid fluctuations of a seemingly random nature. Smooth flows are called *laminar*. The transition form laminar to turbulent flow depends on the Reynolds number and the flow geometry. Very roughly speaking (!), for Re > 10000 flows may be assumed to be turbulent.

The complexity of flows used to be thought surprising, since the physics underlying the governing equations is simply conservation of mass and momentum. Since about 1960, however, it is known that the sweeping generalizations about determinism of Newtonian mechanics made by many scientists (notably Laplace) in the nineteenth century were wrong. Even simple classic nonlinear dynamical systems often exhibit a complicated seemingly random behavior, with such a

sensitivity to initial conditions, that their long-term behavior cannot be predicted in detail. For a discussion of the modern view on (un-)predictability in Newtonian mechanics, see Lighthill (1986).

#### The Stokes equations

Very viscous flows are flows with Re  $\ll 1$ . For Re  $\downarrow 0$  the system (1.25) simplifies to the *Stokes equations*. If we multiply (1.25) by Re and let Re  $\downarrow 0$ , the pressure drops out, which cannot be correct, since we would have four equations (Stokes and mass conservation) for three unknowns  $u_{\alpha}$ . It follows that  $p = \mathcal{O}(\text{Re}^{-1})$ . We therefore substitute

$$p = \text{Re}^{-1} p'$$
 (1.26)

From (1.26) and (1.24) it follows that the dimensional (physical) pressure is  $\mu U p'/L$ . Substitution of (1.26) in (1.25), multiplying by Re and letting Re  $\downarrow 0$  gives the Stokes equations:

$$u_{\alpha,\beta\beta} - p_{,\alpha} = 0. ag{1.27}$$

These linear equations together with (1.10) were solved by Stokes (1851) for flow around a sphere. Surprisingly, the Stokes equations do *not* describe low Reynolds flow *in two dimensions*. This is called the *Stokes paradox*. See Sect. 1.6 of Wesseling (2001) for the equations that govern low Reynolds flows in two dimensions.

The governing equations of incompressible fluid dynamics are given by, if the density is constant, equations (1.25) and (1.10). This is the only situation to be considered in these lecture notes.

Exercise 1.5.1. Derive equation (1.19) from equation (1.18).

**Exercise 1.5.2.** What is the speed of sound and the kinematic viscosity in the air around you? You ride your bike at 18 km/h. Compute your Reynolds number based on a characteristic length (average of body length and width, say) of 1 m. Do you think the flow around you will be laminar or turbulent?

#### 1.6 The convection-diffusion equation

#### Conservation law for material properties

Let  $\varphi$  be a material property, i.e. a scalar that corresponds to a physical property of material particles, such as heat or concentration of a solute in a fluid, for example salt in water. Assume that  $\varphi$  is conserved and can change only through exchange between material particles or through external sources. Let  $\varphi$  be defined per unit of mass. Then the conservation law for  $\varphi$  is:

$$\frac{d}{dt} \int_{V(t)} \rho \varphi dV = \int_{S(t)} \mathbf{f} \cdot \mathbf{n} \, dS + \int_{V(t)} q dV .$$

Here  $\mathbf{f}$  is the *flux vector*, governing the rate of transfer through the surface, and q is the source term. For  $\mathbf{f}$  we assume *Fick's law* (called Fourier's law if  $\varphi$  is temperature):

$$\mathbf{f} = k \nabla \varphi$$
, or  $f_{\alpha} = k \varphi_{,\alpha}$ 

with k the diffusion coefficient. By arguments that are now familiar it follows that

$$\frac{\partial \rho \varphi}{\partial t} + (\rho \varphi u_{\alpha})_{,\alpha} = (k\varphi_{,\alpha})_{,\alpha} + q. \tag{1.28}$$

This is the convection-diffusion equation. The left-hand side represents transport of  $\varphi$  by convection with the flow, the first term at the right represents transport by diffusion.

By using the mass conservation law, equation (1.28) can be written as

$$\rho \frac{D\varphi}{Dt} = (k\varphi_{,\alpha})_{,\alpha} + q. \tag{1.29}$$

If we add a term  $r\varphi$  to the left-hand side of (1.28) we obtain the *convection-diffusion-reaction equation*:

$$\frac{\partial \rho \varphi}{\partial t} + (\rho \varphi u_{\alpha})_{,\alpha} + r \varphi = (k \varphi_{,\alpha})_{,\alpha} + q .$$

This equation occurs in flows in which chemical reactions take place. The *Black-Scholes equation*, famous for modeling option prices in mathematical finance, is also a convection-diffusion-reaction equation:

$$\frac{\partial \varphi}{\partial t} + (1 - k)\varphi_{,x} + k\varphi = \varphi_{,xx}.$$

We will not discuss the convection-diffusion-reaction equation, but only the convection-diffusion equation.

Note that the momentum equation (1.21) comes close to being a convectiondiffusion equation. Many aspects of numerical approximation in computational fluid dynamics already show up in the numerical analysis of the relatively simple convection-diffusion equation, which is why we will devote two special chapters to this equation.

#### Dimensionless form

We can make the convection-diffusion equation dimensionless in the same way as the Navier-Stokes equations. The unit for  $\varphi$  may be called  $\varphi_r$ . It is left as an exercise to derive the following dimensionless form for the convection-diffusion equation (1.28):

$$\frac{\partial \rho \varphi}{\partial t} + (\rho \varphi u_{\alpha})_{,\alpha} = (\text{Pe}^{-1} \varphi_{,\alpha})_{,\alpha} + q , \qquad (1.30)$$

where the *Péclet number* Pe is defined as

$$Pe = \rho_0 U L/k_0$$
.

We see that the Péclet number characterizes the balance between convection and diffusion. For Pe  $\gg 1$  we have dominating convection, for Pe  $\ll 1$  diffusion dominates. If equation (1.30) stands for the heat transfer equation with  $\varphi$  the temperature, then for air we have  $k \approx \mu/0.73$ . Therefore, for the same reasons as put forward in Sect. 1.5 for the Reynolds number, in computational fluid dynamics Pe  $\gg 1$  is the rule rather than the exception.

Exercise 1.6.1. Derive equation (1.30).

#### 1.7 Summary of this chapter

We have introduced Cartesian tensor notation, and have recalled some basic facts from vector analysis. The transport theorem helps to express the conservation laws for mass and momentum of a fluid particle in terms of partial differential equations. This leads to the incompressible Navier-Stokes equations. Nondimensionalization leads to the identification of the dimensionless parameter governing incompressible viscous flows, called the Reynolds number. We have seen that the value of the Reynolds number is usually very high in flows of industrial and environmental interest. We have briefly touched upon the phenomenon of turbulence, which occurs if the Reynolds number is large enough. The convection-diffusion equation, which is the conservation law for material properties that are transported by convection and diffusion, has been derived. Its dimensionless form gives rise to the dimensionless Péclet number, that describes the balance between convection and diffusion.

#### Some self-test questions

Write down the divergence theorem.

What is the total derivative?

Write down the transport theorem.

Write down the governing equations of incompressible viscous flow.

Define the Reynolds number.

Write down the convection-diffusion-reaction equation.

# 2. The stationary convection-diffusion equation in one dimension

#### 2.1 Introduction

Although the one-dimensional case is of no practical use, we will devote a special chapter to it, because important general principles of CFD can be easily analyzed and explained thoroughly in one dimension. We will pay special attention to difficulties caused by a large Péclet number Pe, which is generally the case in CFD, as noted in Sect. 1.6.

In this chapter we consider the one-dimensional stationary version of the dimensionless convection-diffusion equation (1.30) with  $\rho = 1$ :

$$(u\varphi)_{,x} = (\varepsilon\varphi_{,x})_{,x} + q(x) , \quad x \in \Omega \equiv (0,1) ,$$
 (2.1)

where the domain has been chosen to be the unit interval, and  $\varepsilon = 1/\text{Pe}$ . For the physical meaning of this equation, see Sect. 1.6

The purpose of this chapter is:

- To explain that a boundary value problem can be well-posed or ill-posed, and to identify boundary conditions that give a well-posed problem for  $Pe \gg 1$ ;
- To discuss the choice of outflow boundary conditions;
- To explain how the *maximum principle* can tell us whether the exact solution is *monotone*;
- To explain the *finite volume* discretization method;
- To explain the *discrete maximum principle* that may be satisfied by the numerical scheme;
- To study the *local truncation error* on nonuniform grids;
- To show by means of the discrete maximum principle that although the local truncation error is relatively large at the boundaries and in the interior of a nonuniform grid, nevertheless the *global truncation error* can be about as small as on a uniform grid;
- To show how by means of local grid refinement accuracy and computing work can be made independent of the Péclet number;

- To illustrate the above points by numerical experiments;
- To give a few hints about programming in MATLAB.

#### 2.2 Analytic aspects

#### Conservation form

The time-dependent version of (2.1) can be written as

$$\frac{\partial \varphi}{\partial t} = L\varphi + q \; , \quad L\varphi \equiv \left(u\varphi\right)_{,1} - \left(\varepsilon\varphi_{,1}\right)_{,1}.$$

Let us integrate over  $\Omega$ :

$$\frac{d}{dt} \int_{\Omega} \varphi d\Omega = \int_{\Omega} L \varphi d\Omega + \int_{\Omega} q d\Omega.$$

Since

$$\int_{\Omega} L\varphi d\Omega = \left(u\varphi - \varepsilon\varphi_{,1}\right)\Big|_{0}^{1},\tag{2.2}$$

(where we define  $f(x)|_a^b \equiv f(b) - f(a)$ ), we see that

$$\frac{d}{dt}\int_{\varOmega}\varphi d\Omega=\left.\left(u\varphi-\varepsilon\varphi_{,1}\right)\right|_{0}^{1}+\int_{\varOmega}qd\Omega.$$

Hence, if there is no transport through the boundaries x=0,1, and if the source term q=0, then

$$\frac{d}{dt} \int_{\Omega} \varphi d\Omega = 0.$$

Therefore  $\int_{\Omega} \varphi d\Omega$  is conserved. The total amount of  $\varphi$ , i.e.  $\int_{\Omega} \varphi d\Omega$ , can change only in time by transport through the boundaries x=0,1, and by the action of a source term q. Therefore a differential operator such as L, whose integral over the domain  $\Omega$  reduces to an integral over the boundary, is said to be in conservation form.

A famous example of a *nonlinear* convection equation (no diffusion) is the *Burgers equation* (named after the TUD professor J.M. Burgers, 1895–1981):

$$\frac{\partial \varphi}{\partial t} + \frac{1}{2} \left( \varphi^2 \right)_{,1} = 0. \tag{2.3}$$

This equation is in conservation form. But the following version is not in conservation form:

$$\frac{\partial \varphi}{\partial t} + \varphi \varphi_{,1} = 0. \tag{2.4}$$

It is not necessarily wrong to use the nonconservative form of the equation, and many methods are based on this approach. However, when the equation is discretised in nonconservative *integral* form , a sequence of *weak* solutions is not guaranteed to converge to the exact *weak* solution. Weak solutions of general hyperbolic conservation laws are discussed in the course Wi4212 *Advanced Numerical Methods*.

#### An exact solution

Let  $u \equiv 1$ ,  $\varepsilon = \text{constant}$  and q = 0. Then equation (2.1) becomes

$$\varphi_{.1} = \varepsilon \varphi_{.11} , \quad x \in \Omega \equiv (0,1) ,$$
 (2.5)

which can be solved analytically by postulating  $\varphi = Ce^{\lambda x}$ . Substitution in (2.5) shows this is a solution if

$$\lambda - \varepsilon \lambda^2 = 0 \; .$$

hence  $\lambda_1 = 0$  and  $\lambda_2 = 1/\varepsilon$ . Therefore the general solution is

$$\varphi(x) = C_1 + C_2 e^{x/\varepsilon} \,, \tag{2.6}$$

with  $C_{1,2}$  free constants, that must follow from the boundary conditions, in order to determine a unique solution. We see that precisely two boundary conditions are needed, corresponding to the order of the differential equation.

#### **Boundary conditions**

For a second order differential equation, such as (2.1), two boundary conditions are required, to make the solution unique. A differential equation together with its boundary conditions is called a boundary value problem.

We start with the following two boundary conditions, both at x = 0:

$$\varphi(0) = a, \quad \varphi_{.1}(0) = b. \tag{2.7}$$

The first condition, which prescribes a value for  $\varphi$ , is called a *Dirichlet condition*; the second, which prescribes a value for the derivative of  $\varphi$ , is called a *Neumann condition*. The boundary conditions (2.7) are satisfied if the constants in (2.6) are given by  $C_1 = a - \varepsilon b$ ,  $C_2 = \varepsilon b$ , so that the exact solution is given by

$$\varphi(x) = a - \varepsilon b + \varepsilon b e^{x/\varepsilon} . \tag{2.8}$$

#### Ill-posed and well-posed

We now show there is something wrong with boundary conditions (2.7) if  $\varepsilon \ll 1$ . Suppose b is perturbed by an amount  $\delta b$ . The resulting perturbation in  $\varphi(1)$  is

$$\delta\varphi(1) = \varepsilon\delta b e^{1/\varepsilon}$$
.

We see that

$$\frac{|\delta\varphi(1)|}{|\delta b|}\gg 1\quad \text{if}\quad \varepsilon\ll 1\;.$$

Hence, a small change in a boundary condition causes a large change in the solution if  $\varepsilon \ll 1$ . We assume indeed  $\varepsilon \ll 1$ , for reasons set forth in Sect. 1.6. Problems which have large sensitivity to perturbations of the boundary data (or other input, such as coefficients and right-hand side) are called *ill-posed*. Usually, but not always, ill-posedness of a problem indicates a fault in the formulation of the mathematical model. The opposite of ill-posed is *well-posed*. Since numerical

approximations always involve perturbations, ill-posed problems can in general not be solved numerically with satisfactory accuracy, especially in more than one dimension (although there are special numerical methods for solving ill-posed problems with reasonable accuracy; this is a special field). In the present case ill-posedness is caused by wrong boundary conditions.

It is left to the reader to show in Exercise 2.2.2 that the following boundary conditions:

$$\varphi(0) = a, \quad \varphi_{.1}(1) = b.$$
 (2.9)

lead to a well-posed problem. The exact solution is now given by (verify this):

$$\varphi(x) = a + \varepsilon b(e^{(x-1)/\varepsilon} - e^{-1/\varepsilon}). \tag{2.10}$$

Note that equation (2.5) corresponds to a velocity u=1, so that x=0 is an inflow boundary. Hence in (2.9) we have a Dirichlet boundary condition at the inflow boundary and a Neumann boundary condition at the outflow boundary. If we assume u=-1, so that this is the other way around, then the problem is ill-posed as  $\varepsilon \ll 1$  with boundary conditions (2.9). This follows from the result of Exercise 2.2.3. We conclude that it is wrong to give a Neumann condition at an inflow boundary.

Finally, let a Dirichlet condition is given at both boundaries:

$$\varphi(0) = a, \quad \varphi(1) = b. \tag{2.11}$$

The exact solution is

$$\varphi(x) = a + (b - a) \frac{e^{x/\varepsilon} - 1}{e^{1/\varepsilon} - 1}$$
(2.12)

The result of Exercise 2.2.4 shows that boundary conditions (2.11) give a well-posed problem.

To summarize: To obtain a well-posed problem, the boundary conditions must be correct.

#### Maximum principle

We rewrite equation (2.1) as

$$(u\varphi)_1 - (\varepsilon\varphi_{,1})_1 = q(x), \quad x \in \Omega.$$
 (2.13)

Let q(x) < 0,  $\forall x \in \Omega$ . In an interior extremum of the solution in a point  $x_0$  we have  $\varphi_{,1}(x_0) = 0$ , so that

$$\varphi(x_0)u_{.1}(x_0) - \varepsilon\varphi_{.11}(x_0) < 0.$$

Now suppose that  $u_{,1}=0$  (in more dimensions it suffices that  $u_{\alpha,\alpha}=0$ , which is satisfied in incompressible flows). Then  $\varphi_{,11}(x_0)>0$ , so that the extremum cannot be a maximum. This result can be strengthened easily to the case  $q(x) \leq 0$ ,  $\forall x \in \Omega$ : Rewrite (2.13) in the following way and assume  $\Omega = [a, b]$ :

$$L[\varphi] = \frac{u}{\varepsilon} \varphi_{,1} - \varphi_{,11} = \frac{1}{\varepsilon} q(x) \ge 0 \; , \quad x \in \Omega \, . \tag{2.14}$$

with u bounded on  $\Omega$  If we assume  $\varphi(x) \leq M$  in  $\langle a,b \rangle$  and if the maximum M is attained at an interior point  $c \in \langle a,b \rangle$  and there is a d>c,  $\phi(d) < M$  Define z(x) as

$$z(x) = e^{\alpha(x-c)} - 1 \tag{2.15}$$

with  $\alpha > 0$ , a constant to be determined later on. Clearly z(x) has the following properties:

$$z(x) \begin{cases} < 0 \; ; \; a \le x < c \\ = 0 \; ; \quad x = c \\ > 0 \; ; \; c < x \le b \end{cases}$$
 (2.16)

Substituting z(x) in (2.14) gives:

$$L[z] = \frac{u}{\varepsilon} z_{,1} - z_{,11} = \alpha \left(\frac{u}{\varepsilon} - \alpha\right) e^{\alpha(x-c)}$$
(2.17)

Because we assume that u is bounded, we can choose  $\alpha$  such that  $\alpha > \frac{u}{\varepsilon}$  and this will make L[z] < 0. Define

$$w(x) = \phi(x) + \mu z(x), \mu \in \mathbb{R}^+ : \mu < \frac{M - \phi(d)}{z(d)}$$
 (2.18)

Since  $\phi(d) < M$  and z(d) > 0, it is possible to find such a  $\mu$ . Using the fact that  $z(x) < 0, \forall x \in \langle a, c \rangle$  and  $\mu > 0$  it will hold that:

$$x < c \Rightarrow w(x) = \varphi(x) + \mu z(x) < \varphi(x) < M, \tag{2.19}$$

$$x = c \Rightarrow w(x) = \varphi(x) + \mu z(x) = \varphi(x) = M, \tag{2.20}$$

$$x = d \Rightarrow w(x) = \varphi(x) + \mu z(x) < \varphi(x) + M - \varphi(d) < M. \tag{2.21}$$

This means that w(x) attains a maximum  $\overline{M} \geq M$  in  $\langle a,b \rangle$ . But  $L[z] = L[\phi] + \mu L[z] < 0$ . We started our discussion with the proof that when L[z] < 0, z can not attain a maximum at an interior point, so we have a contradiction and the only possibility is that  $\phi \equiv M$ ;  $x \in \langle a,b \rangle$ . For the case that d < c we can redefine z(x) as

$$z(x) = e^{-\alpha(x-c)} - 1, (2.22)$$

and complete the proof in the same way as for the case c < d. Additionally the occurrence of stationary points on the boundary of the interval can be excluded. Hence, if

$$u\varphi_{,1} - \varepsilon\varphi_{,11} \le 0, \quad \forall x \in \Omega,$$

local maxima can occur only at the boundaries. This is called the maximum principle. By reversing signs we see that if  $q(x) \leq 0$ ,  $\forall x \in \Omega$  there cannot be an interior minimum. If  $q(x) \equiv 0$  there cannot be an interior extremum, so that the solution is monotone (in one dimension). The maximum principle gives us important information about the solution, without having to determine the solution. Such information is called a priori information and can help us in the design of numerical methods. Furthermore, take note of the fact that not all PDEs have solutions that conform to the maximum principle. For instance hyperbolic (systems of) equations do not confirm to the maximum principle (wave

equation, instationary Euler equations

If the exact solution has no local maximum or minimum, then "wiggles" (oscillations) in a numerical solution are nor physical, but must be a numerical artifact.

This concludes our discussion of analytic aspects (especially for  $\varepsilon \ll 1$ ) of the convection-diffusion equation. We now turn to numerical solution methods.

**Exercise 2.2.1.** Show that equation (2.3) is in conservation form, and that (2.4) is not.

Exercise 2.2.2. Show that the solution (2.10) satisfies

$$\frac{|\delta\varphi(x)|}{|\delta a|} = 1, \quad \frac{|\delta\varphi(x)|}{|\delta b|} < \varepsilon(1 + e^{-1/\varepsilon}).$$

Hence, the solution is relatively insensitive to the boundary data a and b for all  $\varepsilon > 0$ .

**Exercise 2.2.3.** Show that with u = -1,  $\varepsilon = \text{constant}$  and q = 0 the solution of equation (2.1) is given by

$$\varphi(x) = a + b\varepsilon e^{1/\varepsilon} (1 - e^{-x/\varepsilon})$$
,

so that

$$\frac{\delta \varphi(1)}{\delta b} = \varepsilon (e^{1/\varepsilon} - 1) .$$

Why does this mean that the problem is ill-posed for  $\varepsilon \ll 1$ ?

Exercise 2.2.4. Show that it follows from the exact solution (2.12) that

$$\frac{|\delta\varphi(x)|}{|\delta a|} < 2 \;, \quad \frac{|\delta\varphi(x)|}{|\delta b|} < 1 \;.$$

#### 2.3 Finite volume method

We now describe how equation (2.1) is discretized with the finite volume method. We rewrite (2.1) as

$$L\varphi \equiv (u\varphi)_{,1} - (\varepsilon\varphi_{,1})_{,1} = q, \quad x \in \Omega \equiv (0,1) . \tag{2.23}$$

Let x=0 be an inflow boundary, i.e. u(0)>0. As seen before, it would be wrong to prescribe a Neumann condition (if  $\varepsilon \ll 1$ ), so we assume a Dirichlet condition:

$$\varphi(0) = a . (2.24)$$

Let x = 1 be an outflow boundary, i.e. u(1) > 0. We prescribe either a Neumann condition:

$$\varphi_{.1}(1) = b \,, \tag{2.25}$$

or a Dirichlet condition:

$$\varphi(1) = b. (2.26)$$

The finite volume method works as follows. The domain  $\Omega$  is subdivided in segments  $\Omega_j$ ,  $j=1,\dots,J$ , as shown in the upper part of Fig. 2.1. The segments are called cells or finite volumes or control volumes, and the segment length, denoted by  $h_j$ , is called the mesh size. The coordinates of the centers of the

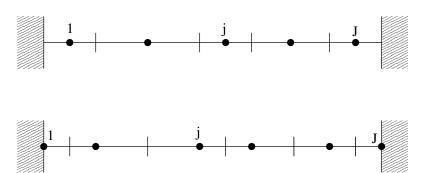


Figure 2.1. Non-uniform cell-centered grid (above) and vertex-centered grid (below).

cells are called  $x_j$ , the size of  $\Omega_j$  is called  $h_j$  and the coordinate of the interface between  $\Omega_j$  and  $\Omega_{j+1}$  is called  $x_{j+1/2}$ . The cell centers are frequently called grid points or nodes. This is called a *cell-centered* grid; the nodes are in the centers of the cells and there are no nodes on the boundaries. In a *vertex-centered* grid one first distributes the nodes over the domain and puts nodes on the boundary; the boundaries of the control volumes are centered between the nodes; see the lower part of Fig. 2.1. We continue with a cell-centered grid. We integrate equation (2.23) over  $\Omega_j$  and obtain:

$$\int_{\Omega_j} L\varphi d\Omega = F|_{j-1/2}^{j+1/2} = \int_{\Omega_j} q d\Omega \cong h_j q_j ,$$

with  $F|_{j-1/2}^{j+1/2} \equiv F_{j+1/2} - F_{j-1/2}$ ,  $F_{j+1/2} = F(x_{j+1/2})$ ,  $F(x) \equiv u\varphi - \varepsilon d\varphi/dx$ . Often, F(x) is called the flux. The following scheme is obtained:

$$L_h \varphi_j \equiv F_{j+1/2} - F_{j-1/2} = h_j q_j , \quad j = 1, \dots, J .$$
 (2.27)

We will call  $u\varphi$  the convective flux and  $\varepsilon d\varphi/dx$  the diffusive flux.

#### Conservative scheme

Summation of equation (2.27) over all cells gives

$$\sum_{j=1}^{J} L_h \varphi_j = F_{J+1/2} - F_{1/2} . \qquad (2.28)$$

We see that only boundary fluxes remain, to that equation (2.28) mimics the conservation property (2.2) of the differential equation. Therefore the scheme (2.27) is called *conservative*. This property is generally beneficial for accuracy and physical realism.

#### Discretization of the flux

To complete the discretization, the flux  $F_{j+1/2}$  has to be approximated in terms of neighboring grid function values; the result is called the *numerical flux*. Central discretization of the convection term is done by approximating the convective flux as follows:

$$(u\varphi)_{j+1/2} \cong u_{j+1/2}\varphi_{j+1/2}, \quad \varphi_{j+1/2} = \frac{1}{2}(\varphi_j + \varphi_{j+1}).$$
 (2.29)

Since u(x) is a known coefficient,  $u_{j+1/2}$  is known. One might think that better accuracy on nonuniform grids is obtained by linear interpolation:

$$\varphi_{j+1/2} = \frac{h_j \varphi_{j+1} + h_{j+1} \varphi_j}{h_j + h_{j+1}} \ . \tag{2.30}$$

Surprisingly, the scheme (2.27) is less accurate with (2.30) than with (2.29), as

we will see. This is one of the important lessons that can be learned from the present simple one-dimensional example.

Upwind discretization is given by

$$(u\varphi)_{j+1/2} \cong \frac{1}{2}(u_{j+1/2} + |u_{j+1/2}|)\varphi_j + \frac{1}{2}(u_{j+1/2} - |u_{j+1/2}|)\varphi_{j+1}.$$
 (2.31)

This means that  $u\varphi$  is biased in upstream direction (to the left for u>0, to the right for u<0), which is why this is called upwind discretization. A complete family of approximations can be defined based on a single parameter  $\kappa$ , the so-called  $\kappa$ -scheme:

$$(u\varphi)_{j+1/2} \cong (2.32)$$

$$u_{j+1/2} \left\{ \frac{(\varphi_j + \varphi_{j+1})}{2} + \frac{1 - \kappa}{4} (-\varphi_{j-1} + 2\varphi_j - \varphi_{j+1}) \right\}, \quad u_{j+1/2} \ge 0$$

$$(u\varphi)_{j+1/2} \cong$$

$$u_{j+1/2} \left\{ \frac{(\varphi_j + \varphi_{j+1})}{2} + \frac{1 - \kappa}{4} (-\varphi_j + 2\varphi_{j+1} - \varphi_{j+2}) \right\}, \quad u_{j+1/2} \le 0$$

A very similar definition is that of the  $\lambda$  scheme:

$$(u\varphi)_{j+1/2} \cong (2.33)$$

$$u_{j+1/2} \left\{ \frac{(\varphi_j + \varphi_{j+1})}{2} - \lambda (\varphi_{j+1} - 2\varphi_j + \varphi_{j-1}) \right\}, \quad u_{j+1/2} \geq 0 (u\varphi)_{j+1/2} \cong$$

$$u_{j+1/2} \left\{ \frac{(\varphi_j + \varphi_{j+1})}{2} - \lambda (\varphi_j - 2\varphi_{j+1} + \varphi_{j+2}) \right\}, \quad u_{j+1/2} \geq 0$$

Important values of these parameters are given in Table The diffusive part of the flux is approximated by

$$(\varepsilon\varphi)_{,1}|_{j+1/2} \cong \varepsilon_{j+1/2}(\varphi_{j+1}-\varphi_j)/h_{j+1/2}, \quad h_{j+1/2} = \frac{1}{2}(h_j+h_{j+1}). \quad (2.34)$$

Scheme	$\kappa$	$\lambda$
Central	1	0
QUICK	$\frac{1}{2}$	$\frac{1}{8}$
Second order upwind	-1	$\frac{1}{2}$
Third order upwind	$\frac{1}{3}$	$\frac{1}{6}$

**Table 2.1.** Important values of the parameters  $\kappa/\lambda$  for the convection schemes (2.32) and (2.33).

#### **Boundary conditions**

At x = 0 we cannot approximate the diffusive flux by (2.34), since the node  $x_0$  is missing. We use the Dirichlet boundary condition, and write:

$$(\varepsilon\varphi)_{,1}|_{1/2} \cong 2\varepsilon_{1/2}(\varphi_1 - a)/h_1. \tag{2.35}$$

This is a one-sided approximation of  $(\varepsilon\varphi_{,1})_{1/2}$ , which might impair the accuracy of the scheme. We will investigate later whether this is the case or not. The convective flux becomes simply

$$(u\varphi)_{1/2} \cong u_{1/2}a$$
. (2.36)

Next, consider the boundary x=1. Assume we have the Neumann condition (2.25). The diffusive flux is given directly by the Neumann condition:

$$(\varepsilon\varphi)_{.1\ J+1/2} \cong \varepsilon_{J+1/2}b. \tag{2.37}$$

Since x=1 is assumed to be an outflow boundary, we have  $u_{J+1/2} > 0$ , so that for the upwind convective flux (2.31) an approximation for  $\varphi_{J+1/2}$  is not required. For the central convective fluxes (2.29) or (2.30) we approximate  $\varphi_{J+1/2}$  with extrapolation, using the Neumann condition:

$$\varphi_{J+1/2} \cong \varphi_J + h_J b/2 \ . \tag{2.38}$$

The Dirichlet condition (2.26) is handled in the same way as at x = 0.

#### The numerical scheme

The numerical flux as specified above can be written as

$$F_{j+1/2} = \beta_j^0 \varphi_j + \beta_{j+1}^1 \varphi_{j+1}, \quad j = 1, \dots, J-1,$$
  

$$F_{1/2} = \beta_1^1 \varphi_1 + \gamma_0, \quad F_{J+1/2} = \beta_J^0 \varphi_J + \gamma_1,$$
(2.39)

where  $\gamma_{0,1}$  are known terms arising from the boundary conditions. For example, for the upwind scheme we obtain the results specified in Exercise 2.3.2.

For future reference, we also give the coefficients for the central schemes. For the central scheme (2.29) we find:

$$\beta_{j}^{0} = \frac{1}{2} u_{j+1/2} + (\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J - 1,$$

$$\beta_{j+1}^{1} = \frac{1}{2} u_{j+1/2} - (\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J - 1, \quad \beta_{1}^{1} = -2\varepsilon_{1/2}/h_{1},$$

$$\gamma_{0} = (u_{1/2} + 2\varepsilon_{1/2}/h_{1})a,$$

$$\beta_{J}^{0} = u_{J+1/2}, \quad \gamma_{1} = u_{J+1/2}h_{J}b/2 - \varepsilon_{J+1/2}b \quad \text{(Neumann)},$$

$$\beta_{J}^{0} = 2\varepsilon_{J+1/2}/h_{J}, \quad \gamma_{1} = (u_{J+1/2} - 2\varepsilon_{J+1/2}/h_{J})b \quad \text{(Dirichlet)}.$$
(2.40)

For the central scheme (2.30) we find:

$$\beta_j^0 = \frac{h_{j+1}}{2h_{j+1/2}} u_{j+1/2} + (\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J-1,$$

$$\beta_{j+1}^1 = \frac{h_j}{2h_{j+1/2}} u_{j+1/2} - (\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J-1.$$
(2.41)

The other coefficients (at the boundaries) are the same as in equation (2.40). On a uniform grid the central schemes (2.40) and (2.41) are identical.

Substitution of equations (2.39)–(2.41) in equation (2.27) gives the following linear algebraic system:

$$L_h \varphi_j = \alpha_j^{-1} \varphi_{j-1} + \alpha_j^0 \varphi_j + \alpha_j^1 \varphi_{j+1} = \tilde{q}_j, \quad j = 1, \dots, J,$$
 (2.42)

with  $\alpha_1^{-1} = \alpha_J^1 = 0$ . This is called the *numerical scheme* or the *finite volume scheme*. Its coefficients are related to those of the numerical flux (2.80) –(2.41) by

$$\alpha_{j}^{-1} = -\beta_{j-1}^{0} , \quad j = 2, \cdots, J ,$$

$$\alpha_{j}^{0} = \beta_{j}^{0} - \beta_{j}^{1} , \quad j = 1, \cdots, J ,$$

$$\alpha_{i}^{1} = \beta_{i+1}^{1} , \quad j = 1, \cdots, J - 1 .$$
(2.43)

The right-hand side is found to be

$$\tilde{q}_j = h_j q_j , \quad j = 2, \cdots, J - 1 ,$$

$$\tilde{q}_1 = h_1 q_1 + \gamma_0 , \quad \tilde{q}_J = h_J q_J - \gamma_1 .$$
(2.44)

#### Stencil notation

The general form of a linear scheme is

$$L_h \varphi_j = \sum_{k \in K} \alpha_j^k \varphi_{j+k} = \tilde{q}_j , \qquad (2.45)$$

with K some index set. For example, in the case of (2.45),  $K = \{-1, 0, 1\}$ . The stencil  $[L_h]$  of the operator  $L_h$  is a tableau of the coefficients of the scheme of the following form:

$$[L_h]_j = [\alpha_j^{-1} \quad \alpha_j^0 \quad \alpha_j^1]. \tag{2.46}$$

We will see later that this is often a convenient way to specify the coefficients. Equation (2.45) is the stencil notation of the scheme.

#### The matrix of the scheme

In matrix notation the scheme can be denoted as

$$Ay = b$$
,  $y = \begin{bmatrix} \varphi_1 \\ \vdots \\ \varphi_J \end{bmatrix}$ ,  $b = \begin{bmatrix} \tilde{q}_1 \\ \vdots \\ \tilde{q}_J \end{bmatrix}$ , (2.47)

where A is the following tridiagonal matrix:

$$A = \begin{bmatrix} \alpha_1^0 & \alpha_1^1 & 0 & \cdots & 0 \\ \alpha_2^{-1} & \alpha_2^0 & \alpha_2^1 & & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & \alpha_{J-1}^{-1} & \alpha_{J-1}^0 & \alpha_{J-1}^1 \\ 0 & \cdots & 0 & \alpha_J^{-1} & \alpha_J^0 \end{bmatrix}.$$
 (2.48)

#### Vertex-centered grid

In the interior of a vertex-centered grid the finite volume method works just as in the cell-centered case, so that further explanation is not necessary. But at the boundaries the procedure is a little different. If we have a Dirichlet condition, for example at x = 0, then an equation for  $\varphi_1$  is not needed, because  $\varphi_1$  is prescribed ( $x_1$  is at the boundary, see Fig. 2.1). Suppose we have a Neumann condition at x = 1. Finite volume integration over the last control volume (which has  $x_J$  as the right end point, see Fig. 2.1) gives:

$$L_h \varphi_J \equiv F_J - F_{J-1/2} = h_J q_J ,$$

where we approximate  $F_J$  as follows, in the case of the central scheme for convection, for example:

$$F_J = u_J \varphi_J - \varepsilon_J b$$
,

where b is given in (2.25).

#### Symmetry

When the velocity  $u \equiv 0$ , the convection-diffusion equation reduces to the diffusion equation, also called heat equation. According to equations (2.39)—(2.41) we have in this case  $\beta_j^1 = -\beta_{j-1}^0$ , so that (2.43) gives  $\alpha_j^1 = \alpha_{j+1}^{-1}$ , which makes the matrix A symmetric. This holds also in the vertex-centered case. Symmetry can be exploited to save computer memory and to make solution methods more efficient. Sometimes the equations are scaled to make the coefficients of size  $\mathcal{O}(1)$ ; but this destroys symmetry, unless the same scaling factor is used for every equation.

#### Two important questions

The two big questions asked in the numerical analysis of differential equations are:

- How well does the numerical solution approximate the exact solution of equation (2.23)?
- How accurately and efficiently can we solve the linear algebraic system (2.47)? These questions will come up frequently in what follows. In the ideal case one shows theoretically that the numerical solution converges to the exact solution as the mesh size  $h_j \downarrow 0$ . In the present simple case, where we have the exact solution (2.6), we can check convergence by numerical experiment.

#### Numerical experiments on uniform grid

We take u=1,  $\varepsilon$  constant, q=0 and the grid cell-centered and uniform, with  $h_j=h=1/12$ . We choose Dirichlet boundary conditions (2.11) with a=0.2, b=1. The exact solution is given by (2.12). The numerical results in this section have been obtained with the MATLAB code cd1. Fig. 2.2 gives results for two values of the Péclet number (remember that  $\varepsilon=1/\text{Pe}$ ). We see

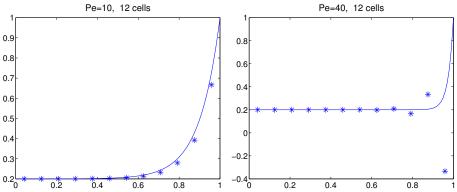


Figure 2.2. Exact solution (—) and numerical solution (\*).

a marked difference between the cases Pe=10 and Pe=40. The numerical solution for Pe=40 is completely unacceptable. We will now analyze why this is so and look for remedies.

#### The maximum principle

In Sect. 2.2 we saw that according to the maximum principle, with q=0 the solution of equation (2.23) cannot have local extrema in the interior. This is confirmed of course in Fig. 2.2. However, the numerical solution for Pe = 40 shows local extrema. These undesirable numerical artifacts are often called "wiggles". It is desirable that the numerical scheme satisfies a similar maximum principle as the differential equation, so that artificial wiggles are excluded. This is the case for *positive schemes*, defined below. Let the scheme be written

in stencil notation:

$$L_h \varphi_j = \sum_{k \in K} \alpha_j^k \varphi_{j+k} = \tilde{q}_j , \quad j = 1, \cdots, J .$$

**Definition 2.3.1.** The operator  $L_h$  is of positive type if

$$\sum_{k \in K} \alpha_j^k = 0, \quad j = 2, \cdots, J - 1$$
 (2.49)

and

$$\alpha_j^k < 0, \quad k \neq 0, \quad j = 2, \dots, J - 1.$$
 (2.50)

Note that a condition is put on the coefficients only in the interior. The following theorem says that schemes of positive type satisfy a similar maximum principle as the differential equation.

Theorem 2.3.1. Discrete maximum principle.

If  $L_h$  is of positive type and

$$L_h \varphi_j \leq 0, \quad j = 2, \cdots, J - 2,$$

then  $\varphi_j \leq \max\{\varphi_1, \varphi_J\}.$ 

Corollary Let conditions (2.49) and (2.50) also hold for j = J. Then  $\varphi_j \leq \varphi_1$ .

A formal proof is given in Sect. 4.4 of Wesseling (2001), but it is easy to see that the theorem is true. Let  $K = \{-1, 0, 1\}$ . We have for every interior grid point  $x_j$ :

$$\varphi_j \le w_{-1}\varphi_{j-1} + w_1\varphi_{j+1} \;, \quad w_{\pm 1} \equiv -\alpha_j^{\pm 1}/\alpha_j^0 \;.$$

Since  $w_{-1}+w_1=1$  and  $w_{\pm 1}>0$ ,  $\varphi_j$  is a weighted average of its neighbors  $\varphi_{j-1}$  and  $\varphi_{j+1}$ . Hence, either  $\varphi_j<\max\{\varphi_{j-1},\varphi_{j+1}\}$  or  $\varphi_j=\varphi_{j-1}=\varphi_{j+1}$ .

Let us now see whether the scheme used for Fig. 2.2 is of positive type. Its stencil is given by equation (2.81) in the exercises at the end of the chapter:

$$[L_h] = \left[ -\frac{1}{2}u - \frac{\varepsilon}{h} \qquad 2\frac{\varepsilon}{h} \qquad \frac{1}{2}u - \frac{\varepsilon}{h} \right] . \tag{2.51}$$

We see that this scheme is of positive type if and only if

$$p < 2 \; , \quad p \equiv \frac{|u|h}{\varepsilon} \; .$$
 (2.52)

The dimensionless number p is called the *mesh Péclet number*.

For the left half of Fig. 2.2 we have p=10/12<2, whereas for the right half p=40/12>2, which explains the wiggles. In general,  $\text{Pe}=UL/\varepsilon$  and  $p=Uh/\varepsilon$ , so that p=Peh/L, with L the length of the domain  $\Omega$  and U representative of the size of u, for example  $U=\max[u(x):x\in\Omega]$ . and for p<2 we must choose h small enough: h/L<2/Pe. Since in practice Pe is usually very large, as shown in Sect. 1.6, this is not feasible (certainly not in more than one dimension), due to computer time and memory limitations. Therefore a scheme

is required that is of positive type for all values of Pe. Such a scheme is obtained if we approximate the convective flux  $u\varphi$  such that a non-positive contribution is made to  $\alpha_j^{\pm 1}$ . This is precisely what the upwind scheme (2.31) is about. For the problem computed in Fig. 2.2 its stencil is, since u>0:

$$[L_h] = \left[ -u - \frac{\varepsilon}{h} \qquad u + 2\frac{\varepsilon}{h} \qquad -\frac{\varepsilon}{h} \right] . \tag{2.53}$$

It is easy to see that this scheme is of positive type for all Pe. Results are given in Fig. 2.3. We see that wiggles are absent, and that the numerical solution satisfies

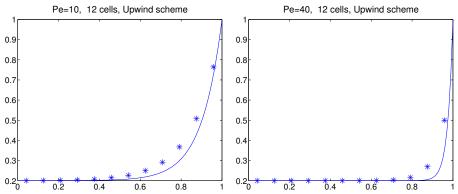


Figure 2.3. Exact solution (—) and numerical solution (\*).

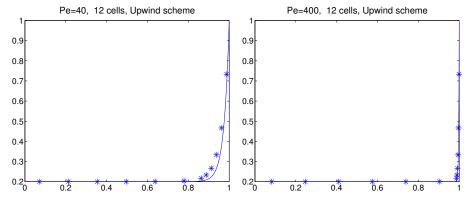
the maximum principle. But the solution is smeared near the outflow boundary. It is as if the numerical solution has a smaller Péclet number than the exact solution. This is because the upwind scheme introduces numerical diffusion; the viscosity is increased with an artificial viscosity coefficient  $\varepsilon_a = uh/2$ . To see this, just replace  $\varepsilon$  by  $\varepsilon + \varepsilon_a$  in the stencil of the central scheme (2.51): it becomes identical to the stencil of the upwind scheme (2.53).

#### Local grid refinement

The preceding figures show for Pe = 40 a rapid variation of the exact solution in a narrow zone near the outflow boundary x = 1. This zone is called a *boundary layer*. From the exact solution (2.13) it follows that the boundary layer thickness  $\delta$  satisfies

$$\delta = \mathcal{O}(\varepsilon) = \mathcal{O}(\text{Pe}^{-1}).$$
 (2.54)

(Landau's order symbol  $\mathcal{O}$  is defined later in this section). We will see later how to estimate the boundary layer thickness when an exact solution is not available. It is clear that to have reasonable accuracy in the boundary layer, the local mesh size must satisfy  $h < \delta$ , in order to have sufficient resolution (i.e. enough grid points) in the boundary layer. This is not the case in the right parts of the preceding figures. To improve the accuracy we refine the grid locally in the boundary layer. We define  $\delta \equiv 6\varepsilon$ ; the factor 6 is somewhat arbitrary and has been determined by trial and error. We put 6 equal cells in  $(0, 1 - \delta)$  and 6 equal cells in  $(1 - \delta, 1)$ . The result is shown in Fig. 2.4. Although the total number of cells remains the same, the accuracy of the upwind scheme has improved



**Figure 2.4.** Exact solution (—) and numerical solution (\*) with local grid refinement; upwind scheme

significantly. Even for Pe = 400, in which case the boundary layer is very thin, the accuracy is good.

Fig. 2.5 gives results for the central scheme (2.29). Surprisingly, the wiggles

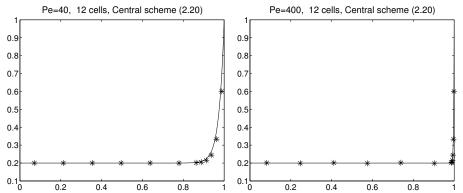


Figure 2.5. Exact solution (—) and numerical solution (\*) with local grid refinement; central scheme (2.29)

which destroyed the accuracy in Fig. 2.2 have become invisible. In the refinement zone the local mesh Péclet number satisfies p=1, which is less than 2, so that according to the maximum principle there can be no wiggles in the refinement zone (see equation (2.52) and the discussion preceding (2.52)). However, inspection of the numbers shows that small wiggles remain outside the refinement zone.

Fig. 2.6 gives results for the central scheme (2.30). This scheme might be expected to be more accurate than central scheme (2.29), because linear interpolation to approximate  $\varphi_{j+1/2}$  is more accurate than averaging on a nonuniform grid. However, we see that for Pe = 400 the opposite is true! Clearly, we are in need of theoretical error analysis. This subject will be touched upon later. A preliminary explanation is as follows. Let the boundary of the refinement zone be located between the nodes  $x_j$  and  $x_{j+1}$ . Call the mesh size inside and outside the refinement zone h and H, respectively. The stencil of the central

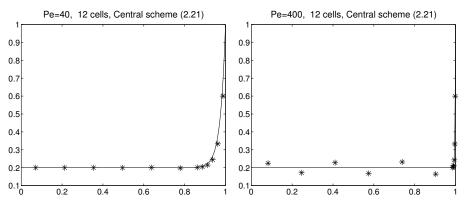


Figure 2.6. Exact solution (—) and numerical solution (\*) with local grid refinement; central scheme (2.30)

scheme scheme (2.29) at  $x_i$  follows from equation (2.40) as:

$$[L_h] = \left[ -\frac{1}{2} - \frac{\varepsilon}{H} \qquad \frac{\varepsilon}{H} + \frac{2\varepsilon}{h+H} \qquad \frac{1}{2} - \frac{2\varepsilon}{h+H} \right] . \tag{2.55}$$

For the central scheme (2.30) we find from equation (2.41):

$$[L_h] = \left[ -\frac{1}{2} - \frac{\varepsilon}{H} - \frac{1}{2} + \frac{h}{h+H} + \frac{\varepsilon}{H} + \frac{2\varepsilon}{h+H} - \frac{H}{h+H} - \frac{2\varepsilon}{h+H} \right]. \tag{2.56}$$

According to definition 2.3.1, one of the necessary conditions for a positive scheme scheme is that the third element in the above stencils is non-positive. For  $\varepsilon \ll 1$  (and consequently  $h/H \ll 1$ ) this element is about 1/2 in (2.55) and 1 in (2.56), which is worse. Furthermore, in (2.56) the central element is negative. We see that (2.56) deviates more from the conditions for a positive scheme than (2.55), so that it is more prone to wiggles.

# Péclet-uniform accuracy and efficiency

The maximum norm of the error  $e_i \equiv \varphi(x_i) - \varphi_i$  is defined as

$$||e||_{\infty} \equiv \max\{|e_i|, \quad j=1,\cdots,J\},$$

where  $\varphi(x)$  is the exact solution. Table 2.2 gives results. The number of cells is the same as in the preceding figures. For Pe = 10 a uniform grid is used, for the other cases the grid is locally refined, as before. We see that our doubts about

Scheme	Pe=10	Pe=40	Pe=400	Pe=4000
Upwind	.0785	.0882	.0882	.0882
Central $(2.20)$	.0607	.0852	.0852	.0852
Central (2.21)	.0607	.0852	.0856	.3657

Table 2.2. Maximum error norm; 12 cells.

the central scheme (2.30) are confirmed. For the other schemes we see that  $||e||_{\infty}$ 

is almost independent of Pe. This is due to the adaptive (i.e. Pe-dependent) local grid refinement in the boundary layer. Of course, since the number of cells J required for a given accuracy does not depend on Pe, computing work and storage are also independent of Pe. We may conclude that computing cost and accuracy are uniform in Pe.

This is an important observation. A not uncommon misunderstanding is that numerical predictions of high Reynolds (or Péclet in the present case) number flows are inherently untrustworthy, because numerical discretization errors ('numerical viscosity') dominate the small viscous forces. This is not true, provided appropriate measures are taken, as was just shown. Local grid refinement in boundary layers enables us to obtain accuracy independent of the Reynolds number. Because in practice Re (or its equivalent such as the Péclet number) is often very large (see Sect. 1.5) it is an important (but not impossible) challenge to realize this also in more difficult multi-dimensional situations. An analysis of Pe-uniform accuracy for a two-dimensional singular perturbation problem will be given in Sect. 3.3.

#### Global and local truncation error

By using Taylor's formula (see below) it is easy to see that the numerical flux as specified above (equations (2.29-2.34)) approaches the exact flux as the grid is refined, *i.e.* as

$$\Delta \downarrow 0, \quad \Delta \equiv \max\{h_j, j = 1, \cdots, J\}$$
 (2.57)

But does this mean that the difference between the numerical and exact solution goes to zero? Surprisingly, this is no simple matter, but one of the deepest questions in numerical analysis. We will present only some basic considerations. We define

**Definition 2.3.2.** Global truncation error

The *global truncation error* is defined as

$$e_j \equiv \varphi(x_j) - \varphi_j , \quad j = 1, \cdots, J ,$$

with  $\varphi(x)$  the exact solution.

Truncation errors are errors that are caused by truncation (to truncate means to shorten by cutting off) of an infinite process. The process we have in mind here is the limit  $\Delta \downarrow 0$ ; we stop at a finite value of  $\Delta$ . Rounding errors are the errors that are caused by the finite precision approximation of real numbers in computer memories. In the numerical approximation of differential equations these are usually much smaller than truncation errors. Here we just assume zero rounding error.

Obviously, the global truncation error is what we are after, but it cannot be estimated directly, because the exact solution is not available. Therefore a quantity is introduced that can be estimated, namely

# **Definition 2.3.3.** Local truncation error

The local truncation error of the discrete operator  $L_h$  is defined as

$$\tau_i \equiv L_h e_i \ , \quad j = 1, \cdots, J \ . \tag{2.58}$$

The exact solution to the differential equation is not a solution of the discrete differential operator: this mismatch is the local truncation error:

$$\tau_j = L_h[\varphi(x_j) - \varphi_j] = L_h[\varphi(x_j)] - L[\varphi_j] = L_h[\varphi(x_j)], \quad j = 1, \dots, J. \quad (2.59)$$

It follows that  $e = L_h^{-1}\tau$ , with e and  $\tau$  algebraic vectors with elements  $e_j$ ,  $\tau_j$ . Hence

$$||e|| \le ||L_b^{-1}|| ||\tau||$$
.

This suggests that a scheme with smaller  $\|\tau\|$  will have a smaller  $\|e\|$  than a scheme with a larger  $\|\tau\|$ . But this need not be so, because  $L_h$  is different for the two schemes, so that  $\|L_h^{-1}\|$  is different. To improve our insight in accuracy, we will now dive into a somewhat complicated but elementary analysis.

#### Estimate of local truncation error in the interior

The purpose of the following elementary but laborious analysis is to eliminate two common misunderstandings. The first is that the meshwidth of grids should vary smoothly for accuracy; this is not true in general, at least not for positive schemes (Definition 2.3.1). The second is that a large local truncation error at a boundary causes a large global truncation error; this is also not true in general.

We begin with estimating the local truncation error. For simplicity u and  $\varepsilon$  are assumed constant. We select the central scheme for convection. The scheme (2.51) with a Dirichlet condition at x=0 and a Neumann condition at x=1 (cf. equation (2.40)) and defining  $h_{j+1/2}=\frac{1}{2}\left(h_j+h_{j+1}\right)$  (Figure 2.7) can be written as

$$L_{h}\varphi_{1} \equiv \left(\frac{u}{2} + \frac{\varepsilon}{h_{3/2}} + \frac{2\varepsilon}{h_{1}}\right)\varphi_{1} + \left(\frac{u}{2} - \frac{\varepsilon}{h_{3/2}}\right)\varphi_{2} = h_{1}q_{1} + \left(u + \frac{2\varepsilon}{h_{1}}\right)a,$$

$$L_{h}\varphi_{j} \equiv -\left(\frac{u}{2} + \frac{\varepsilon}{h_{j-1/2}}\right)\varphi_{j-1} + \varepsilon\left(\frac{1}{h_{j-1/2}} + \frac{1}{h_{j+1/2}}\right)\varphi_{j}$$

$$+ \left(\frac{u}{2} - \frac{\varepsilon}{h_{j+1/2}}\right)\varphi_{j+1} = h_{j}q_{j}, \qquad j = 2, \cdots, J-1,$$

$$L_{h}\varphi_{J} \equiv -\left(\frac{u}{2} + \frac{\varepsilon}{h_{J-1/2}}\right)\varphi_{J-1} + \left(\frac{u}{2} + \frac{\varepsilon}{h_{J-1/2}}\right)\varphi_{J}$$

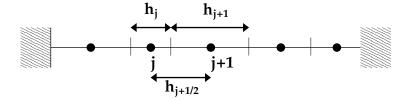
$$= h_{J}q_{J} - \left(\frac{u}{2}h_{J} - \varepsilon\right)b.$$

$$(2.60)$$

Here we made use of the following approximations for the fluxes at the boundary points:

$$F_0 = ua + \frac{2(\varphi_1 - a)\varepsilon}{h_1}$$
 (2.61)

$$F_{J+\frac{1}{2}} = \left(\varphi_J + \frac{bh_J}{2}\right)u + \varepsilon b \tag{2.62}$$



**Figure 2.7.** Definition of  $h_{j+1/2} = \frac{1}{2} (h_j + h_{j+1})$ 

#### Taylor's formula

To estimate the local truncation error we need Taylor's formula:

$$f(x) = f(x_0) + \sum_{k=1}^{n-1} \frac{1}{k!} \frac{d^k f(x_0)}{dx^k} (x - x_0)^k + \frac{1}{n!} \frac{d^n f(\xi)}{dx^n} (x - x_0)^n$$
 (2.63)

for some  $\xi$  between x and  $x_0$ . Of course, f must be sufficiently differentiable. This gives for the exact solution, writing  $\varphi^{(k)}$  for  $d^k\varphi(x)/dx^k$ ,

$$\varphi(x_{j\pm 1}) = \varphi(x_j) \pm h_{j\pm 1/2} \varphi^{(1)}(x_j) + \frac{1}{2} h_{j\pm 1/2}^2 \varphi^{(2)}(x_j) \pm \frac{1}{6} h_{j\pm 1/2}^3 \varphi^{(3)}(x_j) + \frac{1}{24} h_{j\pm 1/2}^4 \varphi^{(4)}(x_j) + \mathcal{O}(h_{j\pm 1/2}^5) ,$$
(2.64)

where  $\mathcal{O}$  is Landau's order symbol, defined as follows:

# Definition 2.3.4. Landau's order symbol

A function  $f(h) = \mathcal{O}(h^p)$  if there exist a constant M independent of h and a constant  $h_0 > 0$  such that

$$\frac{|f(h)|}{h^p} < M, \quad \forall h \in (0, h_0) \ .$$

The relation  $f(h) = \mathcal{O}(h^p)$  is pronounced as "f is of order  $h^p$ ". It is a common misconception in some engineering communities that 'of the order of' gives an indication of the magnitude, whereas it only concerns the asymptotic behavior in the limit  $h \downarrow 0$ .

#### Estimate of local truncation error, continued

We will now see that although we do not know the exact solution, we can nevertheless determine the dependence of  $\tau_j$  on  $h_j$ . We substitute (2.64) in  $L_h(\varphi(x_j))$ , and obtain, after some tedious work that cannot be avoided, for  $j = 2, \dots, J-1$ :

$$L_{h}\varphi(x_{j}) = \tilde{L}_{h}\varphi(x_{j}) + \mathcal{O}(\Delta^{4}) ,$$

$$\tilde{L}_{h}\varphi(x_{j}) \equiv \frac{1}{2}q_{j}(h_{j-1/2} + h_{j+1/2})$$

$$+ \left(\frac{1}{4}u\varphi^{(2)} - \frac{1}{6}\varepsilon\varphi^{(3)}\right)(h_{j+1/2}^{2} - h_{j-1/2}^{2})$$

$$+ \left(\frac{1}{12}u\varphi^{(3)} - \frac{1}{24}\varepsilon\varphi^{(4)}\right)(h_{j+1/2}^{3} + h_{j-1/2}^{3}) ,$$

$$(2.65)$$

where  $\varphi^{(n)} = d^n \varphi(x_j)/dx^n$ . We have

$$\tau_i = L_h e_i = L_h [\varphi(x_i) - \varphi_i] = \tilde{L}_h \varphi(x_i) - h_i q_i + \mathcal{O}(\Delta^4) ,$$

so that we obtain:

$$\tau_{j} = \frac{1}{2} q_{j} \left( h_{j-1/2} - 2h_{j} + h_{j+1/2} \right) 
+ \left( \frac{1}{4} u \varphi^{(2)} - \frac{1}{6} \varepsilon \varphi^{(3)} \right) \left( h_{j+1/2}^{2} - h_{j-1/2}^{2} \right) 
+ \left( \frac{1}{12} u \varphi^{(3)} - \frac{1}{24} \varepsilon \varphi^{(4)} \right) \left( h_{j+1/2}^{3} + h_{j-1/2}^{3} \right) + \mathcal{O}(\Delta^{4}) .$$
(2.66)

The grid is called smooth if the mesh size  $h_j$  varies slowly, or more precisely, if

$$|h_{j+1/2} - h_{j-1/2}| = \mathcal{O}(\Delta^2)$$
 and  $|h_{j-1/2} - 2h_j + h_{j+1/2}| = \mathcal{O}(\Delta^3)$ .

The grid is called rough if the mesh size  $h_i$  varies abruptly, or more precisely, if

$$|h_{i+1/2} - h_{i-1/2}| = \mathcal{O}(\Delta)$$
 and  $|h_{i-1/2} - 2h_i + h_{i+1/2}| = \mathcal{O}(\Delta)$ .

Therefore on smooth grids  $\tau_j = \mathcal{O}(\Delta^3)$ , but on rough grids  $\tau_j = \mathcal{O}(\Delta)$ . Therefore it is often thought that one should always work with smooth grids for better accuracy, but, surprisingly, this is not necessary in general. We will show why later. Note that the locally refined grid used in the preceding numerical experiments is rough, but nevertheless the accuracy was found to be satisfactory.

#### Estimate of local truncation error at the boundaries

For simplicity we now assume the grid uniform, with  $h_j \equiv h$ . Let the scheme be cell-centered. We start with the Dirichlet boundary x = 0. Proceeding as before, we find using Taylor's formula for  $\varphi(x_2)$  in the first equation of (2.60),

$$\begin{split} L_h \varphi(x_1) &= \tilde{L}_h \varphi(x_1) + \mathcal{O}(h^2) \;, \\ \tilde{L}_h \varphi(x_1) &\equiv (u + 2\varepsilon/h) \varphi(x_1) - \varepsilon \varphi^{(1)} + \frac{1}{2} q_1 h \;, \end{split}$$

where  $\varphi^{(1)} = d\varphi(x_1)/dx$ , and  $\varphi(x)$  is the exact solution. We write

$$\tau_1 = L_h[\varphi(x_1) - \varphi_1] = \tilde{L}_h \varphi(x_1) - hq_1 - (u + 2\varepsilon/h)a + \mathcal{O}(h^2)$$
$$= (u + 2\varepsilon/h)[\varphi(x_1) - a] - \varepsilon \varphi^{(1)} - \frac{1}{2}hq_1 + \mathcal{O}(h^2) .$$

We use Taylor's formula for  $a = \varphi(0)$ :

$$a = \varphi(0) = \varphi(x_1) - \frac{1}{2}h\varphi^{(1)} + \frac{1}{8}h^2\varphi^{(2)} + \mathcal{O}(h^3)$$

and find

$$\tau_1 = \frac{h}{4}\varepsilon\varphi^{(2)} + \mathcal{O}(h^2) \ . \tag{2.67}$$

In the interior we have  $\tau_j = \mathcal{O}(h^3)$  on a uniform grid, as seen from equation (2.66). Hence, because in the cell centered case the unknowns do not coincide with the boundary, we have a nonzero local truncation error at the boundary, that is even larger than the local truncation error in the interior domain.... However, it is not necessary to improve the local accuracy near a Dirichlet boundary, which is one of the important messages of this section; we will show this below. But first we will estimate the local truncation error at the Neumann boundary x = 1. By using Taylor's formula for  $\varphi(x_{J-1})$  in the third equation of (2.60) we get

$$\begin{split} L_h \varphi(x_J) &= \tilde{L}_h \varphi(x_J) + \mathcal{O}(h^3) \,, \\ \tilde{L}_h \varphi(x_J) &\equiv \varepsilon \varphi^{(1)} + \frac{1}{2} q_J h - \left( \frac{u}{4} \varphi^{(2)} - \frac{\varepsilon}{6} \varphi^{(3)} \right) h^2 + \mathcal{O}(h^3) \,, \end{split}$$

where  $\varphi^{(n)} = d^n \varphi(x_J)/dx^n$ . We write

$$\tau_{J} = L_{h}[\varphi(x_{J}) - \varphi_{J}] = \tilde{L}_{h}\varphi(x_{J}) - h_{J}q_{J} + (uh_{J}/2 - \varepsilon)b + \mathcal{O}(h^{3})$$
  
=  $\varepsilon\varphi^{(1)} - q_{J}h/2 - (u\varphi^{(2)}/4 - \varepsilon\varphi^{(3)}/6)h^{2} + (uh/2 - \varepsilon)b + \mathcal{O}(h^{3})$ .

We use Taylor's formula for  $b = d\varphi(1)/dx$ :

$$b = \varphi^{(1)} + \frac{1}{2}h\varphi^{(2)} + \frac{1}{8}h^2\varphi^{(3)} + \mathcal{O}(h^3)$$

and find

$$\tau_J = \frac{1}{24} \varepsilon \varphi^{(3)} h^2 + \mathcal{O}(h^3) .$$
(2.68)

# Error estimation with the maximum principle

The student is not expected to be able to carry out the following error analysis independently. This analysis is presented merely to make our assertions about accuracy on rough grids and at boundaries really convincing. We will use the maximum principle to derive estimates of the global truncation error from estimates of the local truncation error.

By e < E we mean  $e_j < E_j$ ,  $j = 1, \dots, J$  and by |e| we mean the grid function with values  $|e_j|$ . We recall that the global and local truncation error are related by

$$L_h e_j = \tau_j, \quad j = 1, \cdots, J. \tag{2.69}$$

Suppose we have a grid function  $E_j$ , which will be called a barrier function, such that

$$L_h E_j \ge |\tau_j|, \quad j = 1, \cdots, J.$$
 (2.70)

We are going to show:  $|e| \leq E$ . From (2.69) and (2.70) it follows that

$$L_h(\pm e_j - E_j) \le 0, \quad j = 1, \dots, J.$$

Let the numerical scheme (2.60) satisfy the conditions of the corollary of Theorem 2.3.1; this is the case if

$$\frac{|u|h_{j+1/2}}{\varepsilon} < 2, \quad j = 1, \dots, J-1.$$
 (2.71)

Then the corollary says

$$\pm e_i - E_i \le \pm e_1 - E_1, \quad j = 2, \dots, J.$$
 (2.72)

Next we show that  $|e_1| \le E_1$ . From  $L_h(\pm e_1 - E_1) \le 0$  it follows (with the use of (2.72) for j = 2) that

$$a(\pm e_1 - E_1) \le b(\pm e_2 - E_2) \le b(\pm e_1 - E_1)$$
,  
 $a = u/2 + 3\varepsilon/h_1$ ,  $b = \varepsilon/h_1 - u/2$ ,

where we assume  $h_2 = h_1$ . Note that 0 < b < a. Therefore  $\pm e_1 - E_1 \le 0$ , hence  $|e_1| \le E_1$ . Substitution in (2.72) results in

$$|e_j| \le E_j, \quad j = 1, \dots, J.$$
 (2.73)

which we wanted to show. It remains to construct a suitable barrier function  $E_j$ . Finding a suitable  $E_j$  is an art.

## Global error estimate on uniform grid

First, assume the grid is uniform:  $h_j = h$ . We choose the barrier function as follows:

$$E_i = M\psi(x_i), \quad \psi(x) \equiv 1 + 3x - x^2,$$
 (2.74)

with M a constant still to be chosen. We find (note that u > 0; otherwise the boundary conditions would be ill-posed for  $\varepsilon \ll 1$ , as seen in Sect. 2.2):

$$\begin{split} L_h\psi(x_1) &= u(1+3h-5h^2/4) + \frac{\varepsilon}{h}(2+3h^2/2) > 2\varepsilon/h \quad \text{for $h$ small enough }, \\ L_h\psi(x_j) &= uh(3-2x_j) + 2\varepsilon h > 2\varepsilon h, \quad j=2,\cdots,J-1 \;, \\ L_h\psi(x_J) &= \varepsilon(1+2h) + uh(1/2+h) > \varepsilon \;. \end{split}$$

According to equations (2.66)–(2.68) there exist constants  $M_1$  ,  $M_2$  ,  $M_3$  such that for h small enough

$$\tau_1 < M_1 h$$
,  
 $\tau_j < M_2 h^3$ ,  $j = 2, \dots, J - 1$ ,  
 $\tau_J < M_3 h^2$ .

Hence, with

$$M = \frac{h^2}{\varepsilon} \max\{M_1/2, M_2/2, M_3\}$$

condition (2.70) is satisfied, so that

$$|e| < E = \mathcal{O}(h^2) .$$

This shows that the fact that the local truncation errors at the boundaries are of lower order than in the interior does not have a bad effect on the global truncation error.

## Global error estimate on nonuniform grid

Next, we consider the effect of grid roughness. From equation (2.66) we see that in the interior

$$\tau_j = \mathcal{O}(\Delta), \quad \Delta = \max\{h_j, \ j = 1, \cdots J\}.$$

We will show that nevertheless  $e = \mathcal{O}(\Delta^2)$ , as for a uniform grid. The barrier function used before does not dominate  $\tau$  sufficiently. Therefore we use the following stratagem. Define the following grid functions:

$$\mu_j^1 \equiv h_j^2 \; , \quad \mu_j^2 \equiv \sum_{k=1}^j h_{k-1/2}^3 \; , \quad \mu_j^3 \equiv \sum_{k=1}^j (h_k^2 + h_{k-1}^2) h_{k-1/2} \; ,$$

where  $h_0 \equiv 0$ . We find with  $L_h$  defined by (2.60) and  $\psi_k(x)$ , k = 1, 2, 3 smooth functions to be chosen later:

$$L_{h}(\psi_{1}(x_{j})\mu_{j}^{1}) = \varepsilon \psi_{1}(x_{j})(-2h_{j+1} + 4h_{j} - 2h_{j-1}) + \qquad (2.75)$$

$$+ \{\varepsilon \frac{d\psi_{1}(x_{j})}{dx} - \frac{1}{2}u\psi_{1}(x_{j})\}(h_{j-1}^{2} - h_{j+1}^{2}) + \mathcal{O}(\Delta^{3}),$$

$$L_{h}(\psi_{2}(x_{j})\mu_{j}^{2}) = \varepsilon \psi_{2}(x_{j})(h_{j-1/2}^{2} - h_{j+1/2}^{2}) + \mathcal{O}(\Delta^{3}), \qquad (2.76)$$

$$L_h(\psi_3(x_j)\mu_j^3) = \varepsilon \psi_3(x_j)(h_{j-1}^2 - h_{j+1}^2) + \mathcal{O}(\Delta^3).$$
 (2.77)

We choose

$$\begin{split} \psi_1 &= -\frac{q(x)}{8\varepsilon} \;, \quad \psi_2 = \frac{1}{6} \varphi^{(3)} - \frac{u}{4\varepsilon} \varphi^{(2)} \;, \\ \psi_3 &= -\frac{d\psi_1}{dx} + \frac{u}{2\varepsilon} \psi_1 \end{split}$$

and define

$$e_j^k \equiv \psi_k(x_j)\mu_j^k \; , \quad k = 1, 2, 3 \; .$$

Remembering (2.69), comparison of (2.75)–(2.77) with (2.66) shows that

$$L_h(e_j - e_j^1 - e_j^2 - e_j^3) = \mathcal{O}(\Delta^3)$$
 (2.78)

The right-hand side is of the same order as the local truncation error in the uniform grid case, and can be dominated by the barrier function (2.74) with  $M = C\Delta^2$ , with C a constant that we will not bother to specify further. For simplicity we assume that  $h_2 = h_1$  and  $h_{J-1} = h_J$ , so that the situation at the boundaries is the same as in the case of the uniform grid. Hence

$$|e_j - e_j^1 - e_j^2 - e_j^3| < C\Delta^2(1 + 3x_j - x_j^2)$$

Since  $e_i^k = \mathcal{O}(\Delta^2)$ , k = 1, 2, 3 we find

$$e_i = \mathcal{O}(\Delta^2) \ . \tag{2.79}$$

which is what we wanted to show. Hence, the scheme defined by (2.60) has second order convergence on arbitrary grids, so that its widespread application is justified. However, recall that the whole proof depends on the numerical scheme being of positive type, and the continuous equation obeying the maximum principle.

#### Vertex-centered grid

On a vertex-centered grid we have grid points on the boundary. Therefore the Dirichlet boundary condition at x=0 gives zero local truncation error, which is markedly better than (2.67). Furthermore, because the cell boundaries are now midway between the nodes, the cell face approximation (2.29) is much more accurate. Indeed, the local truncation error is an order smaller on rough grids for vertex-centered schemes; we will not show this. Therefore it is sometimes thought that vertex-centered schemes are more accurate than cell-centered schemes. But this is not so. In both cases,  $e = \mathcal{O}(\Delta^2)$ . Because this is most surprising for cell-centered schemes, we have chosen to elaborate this case. In practice, both types of grid are widely used.

**Exercise 2.3.1.** Derive equation (2.30). (Remember that linear interpolation is exact for functions of type f(x) = a + bx).

**Exercise 2.3.2.** Assume u > 0. Show that for the upwind scheme the coefficients in the numerical flux are:

$$\beta_{j}^{0} = u_{j+1/2} + (\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J-1,$$

$$\beta_{j+1}^{1} = -(\varepsilon/h)_{j+1/2}, \quad j = 1, \dots, J-1, \quad \beta_{1}^{1} = -2\varepsilon_{1/2}/h_{1},$$

$$\gamma_{0} = (u_{1/2} + 2\varepsilon_{1/2}/h_{1})a, \qquad (2.80)$$

$$\beta_{J}^{0} = u_{J+1/2}, \quad \gamma_{1} = -\varepsilon_{J+1/2}b \quad \text{(Neumann)},$$

$$\beta_{J}^{0} = u_{J+1/2} + 2\varepsilon_{J+1/2}/h_{J}, \quad \gamma_{1} = -2\varepsilon_{J+1/2}b/h_{J} \quad \text{(Dirichlet)}.$$

**Exercise 2.3.3.** Show that with  $\varepsilon$  and u constant on a uniform grid the stencil for the central scheme with Dirichlet boundary conditions is given by

$$[L_h]_1 = \begin{bmatrix} 0 & 3\frac{\varepsilon}{h} + \frac{1}{2}u & \frac{1}{2}u - \frac{\varepsilon}{h} \end{bmatrix},$$

$$[L_h]_j = \begin{bmatrix} -\frac{1}{2}u - \frac{\varepsilon}{h} & 2\frac{\varepsilon}{h} & \frac{1}{2}u - \frac{\varepsilon}{h} \end{bmatrix}, \quad j = 2, \dots, J - 1, \quad (2.81)$$

$$[L_h]_J = \begin{bmatrix} -\frac{1}{2}u - \frac{\varepsilon}{h} & 3\frac{\varepsilon}{h} - \frac{1}{2}u & 0 \end{bmatrix}.$$

**Exercise 2.3.4.** Show that in the refinement zone the local mesh Péclet number satisfies p = 1.

Exercise 2.3.5. Derive equations (2.55) and (2.56).

Exercise 2.3.6. Implement a Neumann boundary condition at x = 1 in the MATLAB program cd1. Derive and implement the corresponding exact solution. Study the error by numerical experiments. Implement wrong boundary conditions: Neumann at inflow, Dirichlet at outflow. See what happens.

**Exercise 2.3.7.** In the program cd1 the size of the refinement zone is del = 6/pe . Find out how sensitive the results are to changes in the factor 6.

**Exercise 2.3.8.** Let  $x_j = jh$  (uniform grid) and denote  $\varphi(x_j)$  by  $\varphi_j$ . Show:

$$(\varphi_j - \varphi_{j-1})/h = \varphi_{,1}(x_j) - \frac{h}{2}\varphi_{,11}(\xi),$$
 (2.82)

$$(\varphi_{j+1} - \varphi_{j-1})/(2h) = \varphi_{,1}(x_j) + \frac{h^2}{6}\varphi_{,111}(\xi),$$
 (2.83)

$$(\varphi_{j-1} - 2\varphi_j + \varphi_{j+1})/h^2 = \varphi_{,11}(x_j) + \frac{h^2}{12}\varphi_{,1111}(\xi) ,$$
 (2.84)

(2.85)

with  $\xi \in [x_{j-1}, x_{j+1}]$ .

Exercise 2.3.9. Show that

$$\frac{\sin x}{\sqrt{x}} = \mathcal{O}(\sqrt{x}).$$

#### Some self-test questions

When is the convection-diffusion equation in conservation form?

Write down the Burgers equation.

When do we call a problem ill-posed?

Formulate the maximum principle.

When is a finite volume scheme in conservation form?

What are cell-centered and vertex-centered grids?

What are the conditions for a scheme to be of positive type? Which desirable property do positive schemes have?

Define the mesh-Péclet number.

Why is it important to have Péclet-uniform accuracy and efficiency?

Derive a finite volume scheme for the convection-diffusion equation.

Derive the condition to be satisfied by the step size h for the central scheme to be of positive type on a uniform grid.

Define the global and local truncation error.

Derive the exact solution of the convection-diffusion equation.

Write down Taylor's formula.

# 3. The stationary convection-diffusion equation in two dimensions

# 3.1 Introduction

We will discuss only new aspects that did not come up in the one-dimensional case. The equation to be studied is the two-dimensional stationary convection-diffusion equation:

$$(u_{\alpha}\varphi)_{,\alpha} - (\varepsilon\varphi_{,\alpha})_{,\alpha} = q(x_1, x_2), \quad (x_1, x_2) \in \Omega \equiv (0, 1) \times (0, 1).$$
 (3.1)

Note that the convection-diffusion equation is a model equation for us that helps us understand the many mathematical properties it shares with the Navier-Stokes equations. We solve for the unknown  $\varphi$ , while the velocity field u is given. Think of  $\varphi$  as the local temperature, or the local concentration of a soluble salt in a fluid.

Suitable boundary conditions are:

$$\varphi = f(x_1, x_2)$$
 on  $\partial \Omega_i$  (Dirichlet), (3.2)

$$\varphi = f(x_1, x_2)$$
 on  $\partial \Omega_o$  (Dirichlet) or (3.3)

$$n_{\alpha}\varphi_{,\alpha} = g(x_1, x_2) \text{ on } \partial\Omega_o \text{ (Neumann)},$$
 (3.4)

where **n** is the outward unit normal on the boundary  $\partial \Omega$ ,  $\partial \Omega_i$  is the inflow boundary (where  $n_{\alpha}\varphi_{,\alpha} < 0$ ) and  $\partial \Omega_o$  is the remainder of  $\partial \Omega$ , to be called the outflow boundary.

We recall that  $\varepsilon = 1/\text{Pe}$ , with the Péclet number  $\text{Pe} \gg 1$ . In the same way as in Sect. 2.2 it can be shown that equation (3.1) is in *conservation form*.

As in one dimension, we have a *maximum principle*. We write (3.1) in the following non-conservative form:

$$u_{\alpha}\varphi_{,\alpha} - (\varepsilon\varphi_{,\alpha})_{,\alpha} = \tilde{q} \equiv q - \varphi u_{\alpha,\alpha} .$$
 (3.5)

If  $\tilde{q} \leq 0$  then local maxima can only occur on the boundary  $\partial \Omega$ . We will not show this here; the interested reader may consult Sect. 2.4 of Wesseling (2001).

#### Purpose of this chapter

The purpose of this chapter is:

- To explain how singular perturbation theory can be used to predict where for  $Pe \gg 1$  thin layers (boundary layers) will occur, and to determine the order of their thickness, without knowing the exact solution;
- To explain the difference between *ordinary and* parabolic boundary layers;
- To show which boundary conditions are suitable for  $Pe \gg 1$ ;
- To introduce the *finite volume method* in two dimensions;
- To show how by means of local grid refinement accuracy and computing work can be made independent of the Péclet number;
- To introduce the *stencil* of the scheme and to show how to generate its coefficient matrix:
- To explain the discrete maximum principle in two dimensions;
- To illustrate the above points by numerical experiments.

Exercise 3.1.1. Show that equation (3.1) is in conservation form by using Theorem 1.2.1.

# 3.2 Singular perturbation theory

Before discussing numerical schemes, we will consider singular perturbation theory for the stationary convection-diffusion equation in two dimensions. In view of our experience in the one-dimensional case, we expect when  $\varepsilon \ll 1$  the occurrence of thin layers in which the solution varies rapidly. Such layers are called boundary layers. As seen in Chapt. 2, local grid refinement is required in boundary layers for accuracy. Therefore it is necessary to know where boundary layers occur and the dependence of their thickness on  $\varepsilon$ . In Chapt. 2 this information was deduced from the exact solution; however, in general the exact solution is not available, of course. But the required information is provided by singular perturbation theory, also called boundary layer theory. The boundary layer concept was first introduced by Ludwig Prandtl in 1904, but the mathematical foundation was developed in the middle of the last century.

# Subcharacteristics

When  $\varepsilon \ll 1$  it is natural to approximate (3.1) by putting  $\varepsilon = 0$ , so that we obtain, switching to nonconservative form:

$$u_{\alpha}\varphi_{,\alpha} = \tilde{q} , \quad \tilde{q} = q - \varphi u_{\alpha,\alpha} .$$
 (3.6)

This is the convection equation. Let us define curves called characteristics in  $\Omega$  space by relations  $x_{\alpha} = x_{\alpha}(s)$ , satisfying

$$x_{\alpha,s} = u_{\alpha} . (3.7)$$

For the derivative along the curve we have

$$\varphi_{,s} = \varphi_{,\alpha} x_{\alpha,s} = u_{\alpha} \varphi_{,\alpha}$$
.

Therefore equation (3.6) reduces to

$$\varphi_{,s} = \tilde{q} \ . \tag{3.8}$$

We see that in the homogeneous case  $\tilde{q}=0$  the solution  $\varphi$  is constant along the characteristics, which is why these curves are important. When  $\varepsilon>0$  we do not have  $\varphi$  constant on the characteristics, but as we will see, these curves still play an important role when  $\varepsilon\ll1$ . To avoid confusion, when  $\varepsilon>0$  the characteristics are called subcharacteristics.

#### A paradox

Let the pattern of streamlines (i.e. (sub)characteristics) be qualitatively as in Fig. 3.1. The characteristic  $C_1$  intersects the boundary in points  $P_1$  and  $P_2$ .

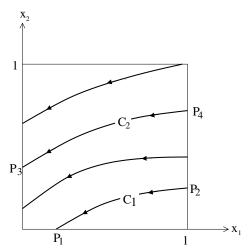


Figure 3.1. Streamline pattern.

Here a boundary condition is given, according to equations (3.2)—(3.4). On  $C_1$  we have  $\varphi = \text{constant} = \varphi(C_1)$ . It is clear that in general  $\varphi(C_1)$  cannot satisfy simultaneously the boundary conditions in  $P_1$  and in  $P_2$ . For example, let  $\varphi(P_1) \neq \varphi(P_2)$  be prescribed by a Dirichlet condition at y = 0 and at x = 1. Do we have  $\varphi(C_1) = \varphi(P_1)$  or  $\varphi(C_1) = \varphi(P_2)$  or  $\varphi(C_1) = (\varphi(P_1) + \varphi(P_2))/2$  or something else? What value to take for  $\varphi(C_1)$ ? The difficulty has to do with the change of type that the partial differential equation (3.1) undergoes when  $\varepsilon = 0$ : for  $\varepsilon > 0$  it is elliptic, for  $\varepsilon = 0$  it is regarded as hyperbolic, because characteristics can be defined in the same way as for second order hyperbolic partial differential equations. It is clear that we cannot get a good approximation to equation (3.1) for  $\varepsilon \downarrow 0$  by simply deleting the small diffusion term. This

paradoxical situation has baffled mathematicians in the nineteenth century, who found the drag of a body in an ideal fluid (zero viscosity) to be zero, whereas in physical experiments the drag around bluff bodies was found to be appreciable, even at very high Reynolds numbers. This was called the paradox of d'Alembert. The problem remains in the nonhomogeneous case  $\tilde{q} \neq 0$ , because the first order equation (3.6) can satisfy only one boundary condition.

Problems that contain a small parameter are called perturbation problems. The terms that are multiplied by the small parameter are regarded as perturbations. If a good approximation can be obtained by simply neglecting the perturbations, we speak of a regular perturbation problem. If a good first approximation cannot be obtained in this way the perturbation problem is called singular. An example of a regular perturbation problem is the sun-earth-moon system. If we neglect the attraction of the moon we still get a good approximation of the orbit and the period of the earth. The above paradox shows that the convection-diffusion equation at large Péclet number is a singular perturbation problem.

We will see that in most cases the flow field is nearly completely described by the inviscid equation, but for very thin regions in the vicinity of the boundary: *The boundary layers*. We will shortly see there are two types of boundary layers: One that corresponds to the boundary layer concept familiar for those with a background in fluid dynamics, but also a second type of boundary layer that results from the specific behavior of the singular perturbation problem.

# Singular perturbation theory

The above paradox is resolved by singular perturbation theory. If we assume flow is from the right to the left in Fig. 3.1, so that  $x_1 = 1$  is an inflow boundary and  $x_2 = 0$  is an outflow boundary, then  $\varphi = \varphi(C_1) = \varphi(P_2)$  is a good approximation for  $\varepsilon \ll 1$  to the solution of (3.1)—(3.4) in  $1 \ge x_2 > \delta = \mathcal{O}(\varepsilon)$  (assuming  $\tilde{q} = 0$ ), whereas (3.6) has to be replaced by a so-called boundary layer equation to obtain an approximation in  $\delta > x_2 \ge 0$ . This can be seen as follows. First, assume that we indeed have  $\varphi(C_1) = \varphi(P_2)$  in  $1 \ge y > \delta$  with  $\delta \ll 1$ . In  $\delta > x_2 \ge 0$  we expect a rapid change of  $\varphi$  from  $\varphi(P_2)$  to  $\varphi(P_1)$ . For derivatives of  $\varphi$  we expect

$$\frac{\partial^m \varphi}{\partial x_2^m} = \mathcal{O}(\delta^{-m}) , \qquad (3.9)$$

so that perhaps the diffusion term in (3.1) cannot be neglected in the boundary layer; this will depend on the size of  $\delta$ . Assume

$$\delta = \mathcal{O}(\varepsilon^{\alpha}) \,\,\,\,(3.10)$$

with  $\alpha$  to be determined. In order to exhibit the dependence of the magnitude of derivatives on  $\varepsilon$  we introduce a *stretched coordinate*  $\tilde{x}_2$ :

$$\tilde{x}_2 = x_2 \varepsilon^{-\alpha} \,, \tag{3.11}$$

which is chosen such that  $\tilde{x}_2 = \mathcal{O}(1)$  in the boundary layer. We take  $\varepsilon$  constant for simplicity. It follows from (3.9)—(3.11) that

$$\frac{\partial^m \varphi}{\partial \tilde{x}_2^m} = \mathcal{O}(1) \tag{3.12}$$

in the boundary layer. In the stretched coordinate, equation (3.1) becomes

$$(u_1\varphi)_1 + \varepsilon^{-\alpha} (u_2\varphi)_{\tilde{2}} - \varepsilon\varphi_{,11} - \varepsilon^{1-2\alpha}\varphi_{,\tilde{2}\tilde{2}} = 0.$$
 (3.13)

Letting  $\varepsilon \downarrow 0$  and using (3.12), equation (3.13) takes various forms, depending on  $\alpha$ . The correct value of  $\alpha$  follows from the requirement, that the solution of the  $\varepsilon \downarrow 0$  limit of equation (3.13) satisfies the boundary condition at  $x_2 = 0$ , and the so-called matching principle.

# Matching principle

As  $\tilde{x}_2$  increases, the solution of (the  $\varepsilon \downarrow 0$  limit of) equation (3.13) has to somehow join up with the solution of (3.6), i.e. approach the value  $\varphi(C_1)$ . In singular perturbation theory this condition is formulated precisely, and is known as the *matching principle*:

$$\lim_{\tilde{x}_2 \to \infty} \varphi_{\text{inner}}(x_1, \tilde{x}_2) = \lim_{x_2 \downarrow 0} \varphi_{\text{outer}}(x_1, x_2) .$$

Here  $\varphi_{\text{inner}}$ , also called the *inner solution*, is the solution of the *inner equation* or boundary layer equation, which is the limit as  $\varepsilon \downarrow 0$  of equation (3.13) for the correct value of  $\alpha$ , which we are trying to determine. Furthermore,  $\varphi_{\text{outer}}$ , also called the *outer solution*, is the solution of the *outer equation*, which is the limit as  $\varepsilon \downarrow 0$  of the original equation, i.e. equation (3.6). The matching principle becomes

$$\varphi_{\text{inner}}(x_1, \infty) = g(x_1) \equiv \lim_{x_2 \downarrow 0} \varphi_{\text{outer}}(x_1, x_2).$$
(3.14)

As already mentioned, the other condition to be satisfied is the boundary condition at  $\tilde{x}_2 = 0$ :

$$\varphi(x_1, 0) = f(x_1) . (3.15)$$

For  $\alpha < 0$  (corresponding to compression rather than stretching) the limit as  $\varepsilon \downarrow 0$  of (3.13) is, taking u constant for simplicity,

$$u\varphi_{,1} = 0 (3.16)$$

so that  $\varphi = \varphi(\tilde{x}_2)$ , which obviously cannot satisfy (3.14), so that the case  $\alpha < 0$  has to be rejected. With  $\alpha = 0$  equation (3.6) is obtained, which cannot satisfy both conditions at  $x_1 = 0$  and  $x_2 = \tilde{x}_2 = 0$ , as we saw.

For  $0 < \alpha < 1$  the limit of (3.13) is, taking  $u_2$  constant for simplicity,

$$u_2\varphi_{,\tilde{2}}=0$$
,

so that the inner solution is independent of  $\tilde{x}_2$ , hence, in general equations (3.14) and (3.15) cannot be satisfied simultaneously. This rules out the case  $0 < \alpha < 1$ .

For  $\alpha = 1$  equation (3.13) becomes as  $\varepsilon \downarrow 0$ :

$$u_2(x_1,0)\varphi_{\tilde{2}} - \varphi_{\tilde{2}\tilde{2}} = 0$$
, (3.17)

where we have used that  $u_2(x_1, x_2) = u_2(x_1, \varepsilon \tilde{x}_2) \to u_2(x_1, 0)$  as  $\varepsilon \downarrow 0$ . The general solution of (3.16) is

$$\varphi = A(x_1) + B(x_1)e^{\tilde{u}_2\tilde{x}_2} , \qquad (3.18)$$

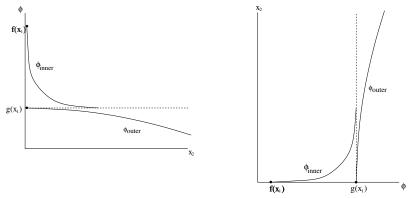
with  $\tilde{u}_2 = u_2(x_1, 0)$ . We can satisfy both (3.14) and (3.15), remembering that we had assumed that y = 0 is an outflow boundary, so that  $\tilde{v} < 0$ . From (3.14) and (3.15) we find

$$A(x_1) = g(x_1), \quad B(x_1) = f(x_1) - g(x_1).$$

This gives us the inner solution. In terms of the unstretched variable  $x_2$  the inner solution is given by

$$\varphi = g(x_1) + [f(x_1) - g(x_1)]e^{\tilde{u}_2 x_2/\varepsilon}$$
.

We see a rapid exponential variation from  $g(x_1)$  to  $f(x_1)$  in a thin layer of thickness  $\delta = \mathcal{O}(\varepsilon)$ , confirming our earlier statement about the behavior of the solution. Fig. 3.2(left) gives a sketch of the inner and outer solutions as a function of  $x_2$  for some given  $x_1$ . An asymptotic approximation for  $\varepsilon \downarrow 0$  that is valid everywhere is given by  $\varphi_{\text{inner}} + \varphi_{\text{outer}} - g(x_1)$  (not shown in the figure). Additionally, Fig. 3.2(right) shows the familiar boundary layer profile we know from fluid dynamics.



**Figure 3.2.** Sketch of inner and outer solutions. The left figure shows the solution as function of the unstretched coordinate  $x_2$ , while the left figure shows the solution in the form of a boundary layer profile

# The distinguished limit

The limit as  $\varepsilon \downarrow 0$  of the stretched equation (3.13) for the special value  $\alpha = 1$  for which the solution of the resulting inner equation can satisfy both the boundary condition and the matching principle is called the *distinguished limit*. In order to show that this limit is unique we will also investigate the remaining values of  $\alpha$  that we did not yet consider, namely  $\alpha > 1$ . Now equation (3.13) gives the following inner equation:

$$\varphi_{\tilde{.}\tilde{2}\tilde{2}} = 0$$
,

with the general solution

$$\varphi = A(x_1) + B(x_1)^{\tilde{z}}$$

The limit of  $\varphi$  as  $\tilde{x}_2 \to \infty$  does not exist, so that the matching principle cannot be satisfied. Hence,  $\alpha = 1$  is the only value that gives a distinguished limit.

The only element of arbitrariness that remains in this analysis is the assumption that we have a boundary layer at  $x_2 = 0$ . Why no boundary layer at  $x_1 = 1$ , and  $\varphi(C_1) = \varphi(P_1)$  (cf. Fig. 3.1)? This can be investigated by assuming a boundary layer at  $x_1 = 1$ , and determining whether a distinguished limit exists or not. This is left as an exercise. It turns out that boundary layers cannot arise at inflow boundaries.

# The role of boundary conditions

The occurrence of boundary layers is strongly influenced by the type of boundary condition. Let (3.15) be replaced by a Neumann condition:

$$-\varphi_{,2}(x_1,0) = f(x_1) . (3.19)$$

As before, a boundary layer of thickness  $\mathcal{O}(\varepsilon)$  is found at  $x_2 = 0$ , and the boundary layer equation is given by (3.16), with general solution (3.18). Taking boundary condition (3.19) into account we find

$$B(x_1) = -\varepsilon f(x_1)/\tilde{u}_2 ,$$

so that  $B(x_1) \to 0$  as  $\varepsilon \downarrow 0$ . Hence, to first order, there is no boundary layer, and the outer solution (solution of (3.6)) is uniformly valid in  $\Omega$ .

## Parabolic and ordinary boundary layers

Those familiar with fluid dynamics may wonder at the boundary layer thickness  $\mathcal{O}(\varepsilon) = \mathcal{O}(1/\text{Pe})$ , since in fluid dynamics laminar boundary layers have thickness  $\mathcal{O}(1/\sqrt{\text{Re}})$ , so that one would have expected  $\delta = \mathcal{O}(1/\sqrt{\text{Pe}})$ . We will now see that the convection-diffusion equation gives rise to two types of boundary layers.

Consider the case that  $x_2 = 0$  is a solid wall, so that  $u_2(x_1, 0) = 0$ . The shape of the characteristics of the outer equation (3.6) might be as in Fig. 3.3, where also  $x_2 = 1$  is assumed to be a solid wall, so that we have a channel flow. Since  $u_2(x_1, 0) = 0$ , the wall  $x_2 = 0$  is a characteristic of the outer equation (3.6) according to (3.7), so that the solution along this characteristic is given by

$$\varphi(x_1, 0) = f_1(0) , \qquad (3.20)$$

assuming  $x_1 = 0$  is a inflow boundary with Dirichlet condition

$$\varphi(0, x_2) = f_1(x_2) . (3.21)$$

Let there also be a Dirichlet condition at the wall y = 0:

$$\varphi(x_1, 0) = f_2(x_1) . \tag{3.22}$$

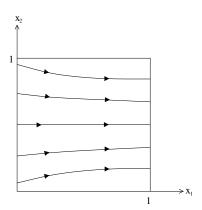


Figure 3.3. Characteristics of equation (3.6) in a channel flow.

This condition cannot in general be satisfied by the outer solution, because it is constant along the wall, which is a characteristic. Hence, we expect a boundary layer at  $x_2 = 0$ . Obviously, this boundary layer will be of different type than obtained before, because the boundary layer solution cannot be given by (3.18), since now we have  $\tilde{u}_2 = 0$ . In order to derive the boundary layer equation, the same procedure is followed as before. Again, we introduce the stretched coordinate (3.11). Keeping in mind that  $u_2 = 0$ , equation (3.1) goes over in

$$(u\varphi)_{.1} - \varepsilon\varphi_{,11} - \varepsilon^{1-2\alpha}\varphi_{,\tilde{2}\tilde{2}} = 0.$$
 (3.23)

The boundary condition is

$$\varphi(x_1, 0) = f_2(x_1) , \qquad (3.24)$$

and the matching principle gives

$$\lim_{\tilde{x}_2 \to \infty} \varphi(x_1, \tilde{x}_2) = \varphi_{\text{outer}}(x_1, 0) . \tag{3.25}$$

Now we take the limit of (3.23) as  $\varepsilon \downarrow 0$ . For  $\alpha < 1/2$  the outer equation at  $x_2 = 0$  is recovered with solution (3.20), which cannot satisfy (3.25). For  $\alpha = 1/2$  the limit of (3.23) is

$$(u\varphi)_{,1} - \varphi_{,\tilde{y}\tilde{y}} = 0 , \qquad (3.26)$$

This is a parabolic partial differential equation, which in general cannot be solved explicitly, but for which it is known that boundary conditions at  $\tilde{y}=0$  and  $\tilde{y}=\infty$  give a well-posed problem. Hence,  $\alpha=1/2$  gives the distinguished limit, and (3.26) is the boundary layer equation. The thickness of this type of boundary layer is  $\mathcal{O}(\sqrt{\varepsilon})$ , which is much larger than for the preceding type, and of the same order as laminar boundary layers in fluid dynamics, for which  $\delta=O(1/\sqrt{\mathrm{Re}})$ .

In order to specify a unique solution for (3.26), in addition a boundary condition has to be specified at x = 0 (assuming u > 0). From (3.21) we obtain the following boundary condition for the boundary layer solution:

$$\varphi(0,\tilde{x}_2) = f_1(\tilde{x}_2\sqrt{\varepsilon})$$
,

which to the present asymptotic order of approximation (we will not go into higher order boundary layer theory) may be replaced by

$$\varphi(0,\tilde{x}_2) = f_1(0) .$$

It is left to the reader to verify that  $\alpha > 1/2$  does not give a distinguished limit.

The cause of the difference between the two boundary layer equations (3.17) (an ordinary differential equation) and (3.26) (a partial differential equation) is the angle which the characteristics of the outer equation (3.6) make with the boundary layer. In the first case this angle is nonzero (cf. Fig. 3.1),in the second case the characteristics do not intersect the boundary layer. The first type is called an *ordinary boundary layer* (the boundary layer equation is an ordinary differential equation), whereas the second type is called a *parabolic boundary layer* (parabolic boundary layer equation).

Summarizing, in the case of the channel flow depicted in Fig. 3.3, for  $\varepsilon \ll 1$  there are parabolic boundary layers of thickness  $\mathcal{O}(\sqrt{\varepsilon})$  at  $x_2 = 0$  and  $x_2 = 1$ , and an ordinary boundary layer of thickness  $\mathcal{O}(\varepsilon)$  at the outflow boundary, unless a Neumann boundary condition is prescribed there.

#### On outflow boundary conditions

It frequently happens that physically no outflow boundary condition is known, but that this is required mathematically. Singular perturbation theory helps to resolve this difficulty. If  $\varepsilon = \mathcal{O}(1)$  such a physical model is incomplete, but for  $\varepsilon \ll 1$  an artificial (invented) outflow condition may safely be used to complete the mathematical model, because this does not affect the solution to any significant extent. Furthermore, an artificial condition of Neumann type is to be preferred above one of Dirichlet type. This may be seen by means of singular perturbation theory, as follows.

Consider the following physical situation: an incompressible flow with given velocity field  $\boldsymbol{u}$  through a channel, the walls of which are kept at a known temperature. We want to know the temperature of the fluid, especially at the outlet. This leads to the following mathematical model. The governing equation is (3.1), with  $\varphi$  the temperature. Assume  $\varepsilon \ll 1$ , and  $u_1 > 0$ . We have  $\varphi$  prescribed at  $x_1 = 0$  and at  $x_2 = 0, 1$ , but at  $x_1 = 1$  we know nothing. Hence, we cannot proceed with solving (3.1), either analytically or numerically. Now let us just postulate some temperature profile at  $x_1 = 1$ :

$$\varphi(1, x_2) = f_3(x_2).$$

An ordinary boundary layer will occur at  $x_1 = 1$ , with solution, derived in the way discussed earlier (cf. equation 3.18), given by

$$\varphi(x_1, x_2) = \varphi_{\text{outer}}(1, x_2) + \{f_3(x_2) - \varphi_{\text{outer}}(1, x_2)\}e^{\tilde{u_1}(x_1 - 1)\varepsilon}, \qquad (3.27)$$

where  $\tilde{u}_1 \equiv u_1(1, x_2)$ . This shows that the invented temperature profile  $f_3(x_2)$  influences the solution only in the thin (artificially generated) boundary layer at  $x_1 = 1$ . This means that the computed temperature outside this boundary layer

will be correct, regardless what we take for  $f_3(x_2)$ . When  $\varepsilon = \mathcal{O}(1)$  this is no longer true, and more information from physics is required, in order to specify  $f_3(x_2)$  correctly. In physical reality there will not be a boundary layer at all at  $x_1 = 1$ , of course. Therefore a more satisfactory artificial outflow boundary condition is

$$\varphi(1,x_2)_{.1}=0$$
,

since with this Neumann boundary condition there will be no boundary layer at  $x_1 = 1$  in the mathematical model.

**Exercise 3.2.1.** Show that there is no boundary layer at  $x_1 = 1$ , if this is an inflow boundary. Hint: choose as stretched coordinate  $\tilde{x}_1 = (x_1 - 1)/\varepsilon^{\alpha}$ .

**Exercise 3.2.2.** Consider equation (2.5). Show that with Dirichlet boundary conditions for  $\varepsilon \ll 1$  there is a boundary layer of thickness  $\mathcal{O}(\varepsilon)$  at  $x_1 = 1$  and not at  $x_1 = 0$ .

Exercise 3.2.3. Derive equation (3.27).

#### 3.3 Finite volume method

#### Problem statement

In this section we study the numerical approximation of the two-dimensional stationary convection-diffusion equation, for convenience written in Cartesian tensor notation:

$$L\varphi \equiv (u_{\alpha}\varphi)_{,\alpha} - (\varepsilon\varphi_{,\alpha})_{,\alpha} = q$$
,  $\alpha = 1, 2$ ,  $(x_1, x_2) \in (0, 1) \times (-1, 1)$ . (3.28)

We assume that we have solid walls at  $x_2 = -1, 1$ , so that  $u_2(x_1, -1) = u_2(x_1, 1) = 0$ . Let  $u_1 < 0$ , so that  $x_1 = 0$  is an outflow boundary. In view of what we learned in Sect. 3.2 we choose a Neumann boundary condition at  $x_1 = 0$ , and Dirichlet boundary conditions at the other parts of the boundary:

$$\begin{array}{rcl} \varphi_{,1}(0,x_2) & = & g^3(x_2) \;,\; \varphi(x_1,-1) \; = \; g^1(x_1) \;, \\ \varphi(x_1,1) & = & g^1(x_1) \;,\; \varphi(1,x_2) \; = \; g^2(x_2) \;. \end{array}$$

This corresponds to the channel flow problem studied before. We assume symmetry with respect to the centerline of the channel, so that we have to solve only in half the domain, i.e. in  $\Omega \equiv (0,1) \times (-1,0)$ . At the centerline our boundary condition is the symmetry condition  $\varphi_{,2}=0$ , so that the boundary conditions are:

$$\varphi_{,1}(0,x_2) = g^3(-x_2), \quad \varphi(x_1,-1) = g^1(x_1), 
\varphi_{,2}(x_1,0) = 0, \qquad \varphi(1,x_2) = g^2(-x_2).$$
(3.29)

The computational domain is illustrated in Fig. 3.4.

As discussed before, it is best to choose at the outflow boundary a homogeneous Neumann condition, *i.e.*  $g^3 \equiv 0$ , but for the purpose of numerical experimentation we leave the possibility of choosing a nonhomogeneous Neumann condition

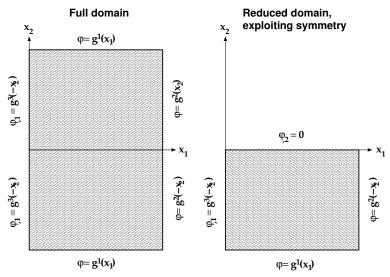


Figure 3.4. The computational domain can be reduced by exploiting symmetry in the geometry and the boundary conditions.

open. We will not discuss the three-dimensional case, because this does not provide new insights.

Our purpose in this section is to show, as in Sect. 2.3, but this time in two dimensions, that (3.28) can be solved numerically such that accuracy and computing cost are uniform in Pe. Therefore, fear that it is impossible to compute high Péclet (Reynolds) number flow accurately is unfounded, as argued before. For simplicity we assume horizontal flow:  $u_2 \equiv 0$ , and we will simply write u instead of  $u_1$ .

# Choice of grid

In view of what we learned in Sect. 3.2, we expect a parabolic (because  $u_2 = 0$ ) boundary layer at  $x_2 = -1$  with thickness  $\mathcal{O}(\sqrt{\varepsilon})$ . If  $g^3 \equiv 0$  we have a homogeneous Neumann condition at the outflow boundary, and there is no boundary layer at  $x_1 = 0$ . Just as in Sect. 2.3, in order to make accuracy and computing work uniform in  $\varepsilon$ , we choose a grid with local refinement in the boundary layer, as sketched in Fig. 3.5. We will apply grid refinement near the outflow boundary later. The region of refinement has thickness  $\sigma = \mathcal{O}(\sqrt{\varepsilon})$ , i.e. the boundary layer thickness scales asymptotically with the inverse of the Péclet number. The precise choice of  $\sigma$  will be discussed later, and is such that the boundary layer falls inside the refinement region. The refined part of the grid is called  $G_f$ , the interface between the refined and unrefined parts is called  $\Gamma$  and the remainder of the grid is called  $G_c$ . The mesh sizes in  $G_f$  and  $G_c$  are uniform, as indicated in Fig. 3.5. Note that the location of the horizontal grid lines depends on  $\varepsilon$ .

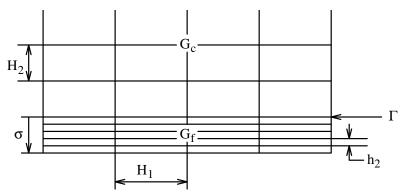


Figure 3.5. Computational grid

#### Finite volume discretization

We choose a cell-centered scheme. The cell centers are labeled by integer twotuples (i, j) in the usual way:  $\Omega_{ij}$  is the cell with center at  $(x_i, y_j)$ . Hence, for example, (i + 1/2, j) refers to the center of a vertical cell edge. Cell-centered finite volume discretization is used as described in Sect. 2.3. For completeness the discretization is summarized below. The finite volume method gives by integration over  $\Omega_{ij}$  and by using the divergence theorem:

$$\int_{\Omega_{ij}} L\varphi d\Omega = \left[ \int_{\mathbf{x}_{i+1/2,j-1/2}}^{\mathbf{x}_{i+1/2,j-1/2}} - \int_{\mathbf{x}_{i-1/2,j-1/2}}^{\mathbf{x}_{i-1/2,j-1/2}} \right] (u\varphi - \varepsilon\varphi_{,1}) dx_{2} 
+ \left[ \int_{\mathbf{x}_{i-1/2,j+1/2}}^{\mathbf{x}_{i+1/2,j-1/2}} - \int_{\mathbf{x}_{i-1/2,j-1/2}}^{\mathbf{x}_{i+1/2,j-1/2}} \right] (-\varepsilon\varphi_{,2}) dx_{1} 
= F^{1} \Big|_{i-1/2,j}^{i+1/2,j} + F^{2} \Big|_{i,j-1/2}^{i,j+1/2}.$$

The approximation of the numerical fluxes  $F^{1,2}$  is given below. The right-hand side of equation (3.28) is numerically integrated over  $\Omega_{ij}$  as follows:

$$\int_{\Omega_{ij}} q d\Omega \cong \tilde{q}_{ij} \equiv H_1 K_j q(\mathbf{x}_{ij}) , \qquad (3.30)$$

where  $K_j$  is the vertical dimension of  $\Omega_{ij}: K_j = h_2$  in  $G_f$  and  $K_j = H_2$  in  $G_c$ . The cells are numbered i = 1, ..., I and j = 1, ..., J in the  $x_1$ - and  $x_2$ -directions, respectively. The following scheme is obtained:

$$L_h \varphi_{ij} \equiv F^1 \Big|_{i-1/2,j}^{i+1/2,j} + F^2 \Big|_{i,j-1/2}^{i,j+1/2} = \tilde{q}_{ij} . \tag{3.31}$$

If we sum (3.31) over all cells only boundary fluxes remain, so that the scheme is *conservative* (cf. Sect. 2.3).

#### The numerical flux

How to approximate the numerical fluxes  $F^{1,2}$  in terms of neighboring grid function values follows directly from the one-dimensional case discussed in the

preceding chapter. With upwind discretization for the first derivative (taking into account that u < 0), the numerical fluxes  $F^{1,2}$  are approximated as follows:

$$F_{i+1/2,j}^{1} \cong K_{j}[u_{i+1/2,j}\varphi_{i+1,j} - \varepsilon(\varphi_{i+1,j} - \varphi_{ij})/H_{1}],$$
  

$$F_{i,j+1/2}^{2} \cong -2H_{1}\varepsilon(\varphi_{i,j+1} - \varphi_{ij})/(K_{j} + K_{j+1}),$$
(3.32)

With the central scheme for the first derivative  $F^1$  becomes:

$$F_{i+1/2,j}^1 \cong K_j[u_{i+1/2,j}(\varphi_{i,j} + \varphi_{i+1,j})/2 - \varepsilon(\varphi_{i+1,j} - \varphi_{ij})/H_1].$$
 (3.33)

The boundary conditions are implemented as in Sect. 2.3. This gives both for the upwind and central schemes:

$$F_{1/2,j}^{1} \cong K_{j}[u_{1/2,j}(\varphi_{1j} - H_{1}g^{4}(y_{j})/2) - \varepsilon g^{4}(y_{j})],$$

$$F_{I+1/2,j}^{1} \cong K_{j}[u_{I+1/2,j}\varphi_{I+1/2,j} - 2\varepsilon(\varphi_{I+1/2,j} - \varphi_{Ij})/H_{1}],$$

$$F_{i,1/2}^{2} \cong -2H_{1}\varepsilon(\varphi_{i,1} - \varphi_{i,1/2})/h_{2},$$

$$F_{i,J+1/2}^{2} = 0,$$

$$(3.34)$$

where  $\varphi_{I+1/2,j}$  and  $\varphi_{i,1/2}$  are given by the boundary conditions (3.29).

#### The stencil of the scheme

Although this is tedious, we will spell out more details of the scheme, as preparation for a MATLAB program. The numerical fluxes as specified above can be written as

$$\begin{split} F_{i+1/2,j}^1 &= \beta_{ij}^0 \varphi_{ij} + \beta_{i+1,j}^1 \varphi_{i+1,j}, \quad i = 1, \cdots, I-1 \;, \quad j = 1, \cdots, J \;, \\ F_{1/2,j}^1 &= \beta_{1j}^1 \varphi_{1,j} + \gamma_j^0 \;, \quad j = 1, \cdots, J \;, \\ F_{I+1/2,j}^1 &= \beta_{1j}^0 \varphi_{Ij} + \gamma_j^1 \;, \quad j = 1, \cdots, J \;, \\ F_{i,j+1/2}^2 &= \beta_{ij}^2 \varphi_{ij} + \beta_{i,j+1}^3 \varphi_{i,j+1}, \quad i = 1, \cdots, I \;, \quad j = 1, \cdots, J-1 \;, \\ F_{i,1/2}^2 &= \beta_{i1}^3 \varphi_{i,1} + \gamma_i^2 \;, \quad i = 1, \cdots, I \;, \\ F_{i,J+1/2}^2 &= 0 \;, \quad i = 1, \cdots, I \;, \end{split} \tag{3.35}$$

where  $\gamma^0, \gamma^1, \gamma^2$  are known terms arising from the boundary conditions. The  $\beta$  and  $\gamma$  coefficients follow easily from (3.34), and will not be written down. The scheme consists of a linear system of equations of the form

$$L_h \varphi_{ij} = \sum_{k=-1}^{1} \sum_{l=-1}^{1} \alpha_{ij}^{kl} \varphi_{i+k,j+l} , \qquad (3.36)$$

where the only nonzero  $\alpha$  coefficients are

$$\alpha_{ij}^{-1,0} = -\beta_{i-1,j}^{0} , \quad \alpha_{ij}^{1,0} = \beta_{ij}^{1}$$

$$\alpha_{ij}^{0,-1} = -\beta_{i,j-1}^{2} , \quad \alpha_{ij}^{0,1} = \beta_{ij}^{3}$$

$$\alpha_{ij}^{0,0} = (\beta^{0} - \beta^{1} + \beta^{2} - \beta^{3})_{ij} .$$
(3.37)

The scheme has a five-point stencil:

$$[L_h] = \left[ \begin{array}{cc} \alpha^{0,1} \\ \alpha^{-1,0} & \alpha^{0,0} \\ \alpha^{0,-1} \end{array} \alpha^{1,0} \right].$$

## The maximum principle

Just as in the one-dimensional case, we have a discrete maximum principle. We generalize Definition 2.3.1 to the two-dimensional case as follows:

**Definition 3.3.1.** The operator  $L_h$  is of positive type if for  $i=2,\cdots,I-1$  and  $j=2,\cdots,J-1$ 

$$\sum_{l,l} \alpha_{ij}^{kl} = 0 \tag{3.38}$$

and

$$\alpha_{ij}^{kl} < 0, \quad (k,l) \neq (0,0) \,.$$
 (3.39)

The following theorem says that schemes of positive type satisfy a similar maximum principle as the differential equation.

Theorem 3.3.1. Discrete maximum principle.

If  $L_h$  is of positive type and

$$L_h \varphi_{ij} \le 0$$
,  $i = 2, \dots, I - 1$ ,  $j = 2, \dots, J - 2$ ,

then  $\varphi_{ij} \leq \max_{ij} \{ \varphi_{1j}, \ \varphi_{Ij}, \ \varphi_{i1}, \ \varphi_{iJ} \}.$ 

In other words, local maxima can only occur in cells adjacent to the boundaries.

It is left to the reader to show that with the upwind scheme  $L_h$  is of positive type, and with the central scheme this is the case if (taking u constant for simplicity) the mesh Péclet number satisfies

$$p < 2 \; , \quad p \equiv \frac{|u|h}{\varepsilon} \; ,$$

just as in the one-dimensional case (cf. equation (2.52)).

# The matrix of the scheme

In matrix notation the scheme (3.31) can be denoted as

$$Ay = b$$
.

Let the algebraic vector y contain the unknowns in lexicographic order:

$$y_m = \varphi_{ij}, \quad m = i + (j-1)I.$$
 (3.40)

Vice-versa, i and j follow from m as follows:

$$j = \text{floor}[(m-1)/I], \quad i = m - (j-1)I,$$
 (3.41)

where floor(a/b) is the largest integer  $\leq a/b$ . The right-hand side b contains  $\tilde{q}_{ij}$  in lexicographic order. The relation between the grid indices i,j and the lexicographic index m is illustrated in Fig. 3.6 With lexicographic ordering, A is an  $IJ \times IJ$  matrix with the following block-tridiagonal structure:

9	10	11	12	1,3	2,3	3,3	4,3
5	6	7	8	1,2	2,2	3,2	4,2
1	2	3	4	1.1	2.1	3.1	4.1

Figure 3.6. Relation between lexicographic and grid indices

$$A = \begin{bmatrix} B_1 & D_1 & 0 & \cdots & 0 \\ C_2 & B_2 & D_2 & & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & & C_{J-1} & B_{J-1} & D_{J-1} \\ 0 & \cdots & 0 & C_J & B_J \end{bmatrix},$$

where  $B_j$  are  $I \times I$  tridiagonal matrices, given by

$$B_{j} = \begin{bmatrix} \alpha_{1j}^{0,0} & \alpha_{1j}^{1,0} & 0 & \cdots & 0 \\ \alpha_{2j}^{-1,0} & \alpha_{2j}^{0,0} & \alpha_{2j}^{1,0} & & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & & \alpha_{I-1,j}^{-1,0} & \alpha_{I-1,j}^{0,0} & \alpha_{I-1,j}^{1,0} \\ 0 & \cdots & 0 & \alpha_{I,j}^{-1,0} & \alpha_{I,j}^{0,0} \end{bmatrix},$$

and  $C_j$  and  $D_j$  are  $I \times I$  diagonal matrices, given by

$$C_j = \text{diag}\{\alpha_{ij}^{0,-1}\}, \quad D_j = \text{diag}\{\alpha_{ij}^{0,1}\}.$$

# Remarks on the MATLAB program cd2

The numerical scheme described above has been implemented in the MATLAB program cd2, available at the author's website; see the Preface.

The student is not expected to fully understand this program, because use is made of somewhat advanced features, such as meshgrid and reshape, in order to avoid for loops. Avoiding for loops is essential for efficiency in MATLAB, as can be seen in the code cd2 by comparing computing time (using tic···toc) for generating the matrices A1 and A3. Generation of A1 is done in a simple way with for loops, and requires 1.22 time units on a  $32 \times 72$  grid, whereas the more sophisticated program for A3 takes 0.067 time units; this discrepancy grows rapidly with increasing grid size. The time used for solving the system Ay = b is 0.32.

In MATLAB, A is generated as a sparse matrix by

```
d = [-I; -1; 0; 1; I];
A = spdiags([a1 a2 -a1-a2-a4-a5 a4 a5], d, n, n);
```

with suitable definition of the diagonals  $\mathtt{a1}\cdots\mathtt{a5}$ . Exploiting sparsity is essential for saving memory and computing time. The solution of the system Ay = b is obtained in MATLAB by

$$y = A b;$$

This means that we solve with a direct method using sparse LU factorization. For large systems, that occur particularly when partial differential equations are solved in three-dimensional domains, direct methods frequently demand intolerable amounts of computer time and memory, even when sparsity is exploited. Efficient solution methods for solving the algebraic systems arising from numerical schemes for partial differential equations will be discussed in Chapt. 6.

#### Numerical experiments

The purpose of the numerical experiments with the program cd2 that we will now describe is to demonstrate, as we did in Sect. 2.3 for the one-dimensional case, that we can achieve Péclet-uniform accuracy and efficiency, and that accurate results can be obtained on grids with large jumps in mesh size. This is shown theoretically in Sect. 4.7 of Wesseling (2001), but here we confine ourselves to numerical illustration.

In order to be able to assess the error, we choose an exact solution. Of course, this solution has to exhibit the boundary layer behavior occurring in practice. We choose the following solution of the boundary layer (inner) equation (3.26):

$$\varphi = \frac{1}{\sqrt{2 - x_1}} \left\{ \exp(-\frac{x_2^2}{4\varepsilon(2 - x_1)}) + \exp(-\frac{(2 - x_2)^2}{4\varepsilon(2 - x_1)}) \right\} .$$

The right-hand side and boundary conditions in (3.29) are chosen accordingly. The exact solution is symmetric with respect to  $x_2 = 1$ , as assumed by the boundary conditions (3.29).

Because the solution is extremely smooth in  $\Omega_c$ , it turns out that in  $G_c$  the number of cells in the vertical direction can be fixed at 4; the maximum of the error is found to always occur in  $G_f$ . We take

$$\sigma = 8\sqrt{\varepsilon} \ . \tag{3.42}$$

Table 3.1 gives results for the cell-centered upwind case. Exactly the same results (not shown) are obtained for  $\varepsilon = 10^{-5}$  and  $\varepsilon = 10^{-7}$ , showing  $\varepsilon$ -uniform accuracy. Of course, computing time and memory are also independent of  $\varepsilon$ , because they depend only on  $nx_1$  and  $nx_2$ . The maximum error is found to occur in the interior of the boundary layer. Because we use the upwind scheme

$nx_1$	$nx_2$	error $*10^4$
8	32	54
32	64	14
128	128	3.6

**Table 3.1.** Maximum error as function of number of grid-cells for  $\varepsilon = 10^{-3}$ ; cell-centered upwind discretization.  $nx_1$ : horizontal number of cells;  $nx_2$ : vertical number of cells in  $G_f$ .

in the  $x_1$ -direction and the central scheme in the  $x_2$ -direction, we expect for the error  $e = \mathcal{O}(H_1 + h_2^2)$ , so that the error should decrease by a factor 4 at each refinement in Table 3.1; this expectation is confirmed. Table 3.2 gives results for central discretization of the convection term. Visual inspection of graphical output (not shown) shows no visible wiggles. But very small wiggles are present.

nx	ny	error $*10^4$
8	16	92
16	32	28
32	64	7.8
64	128	2.1

**Table 3.2.** Cell-centered central discretization;  $\varepsilon = 10^{-3}$ .

These are the cause that the rate of convergence is somewhat worse than the hoped for  $\mathcal{O}(H_1^2 + h_2^2)$ , but here again the same results are obtained for  $\varepsilon = 10^{-7}$ , showing uniformity in  $\varepsilon$ .

We may conclude that in practice work and accuracy can be made to be uniform in  $\varepsilon$ , by suitable local mesh refinement according to Fig. 3.5 and equation (3.42). Hence, in principle, high Reynolds number flows are amenable to computation.

As before, having come to the end of this chapter, looking again at the list of items that we wanted to cover given in Sect. 3.1 will help the reader to remind himself of what the main points were that we wanted to emphasize.

Exercise 3.3.1. Derive equations (3.32) and (3.33).

**Exercise 3.3.2.** Take  $u_1, u_2$  and  $\varepsilon$  constant, and the grid uniform. Discretize the convection-diffusion equation (3.28) with the finite volume method, using the central scheme. Show that the resulting stencil is

$$[L_h] = \varepsilon \begin{bmatrix} \frac{1}{2}p_1 - \frac{h_2}{h_1} & \frac{1}{2}p_2 - \frac{h_1}{h_2} \\ -\frac{1}{2}p_1 - \frac{h_2}{h_1} & 2(\frac{h_1}{h_2} + \frac{h_2}{h_1}) & \frac{1}{2}p_1 - \frac{h_2}{h_1} \\ -\frac{1}{2}p_2 - \frac{h_1}{h_2} & \end{bmatrix},$$

where  $p_1 \equiv u_1 h_1/\varepsilon$  and  $p_2 \equiv u_2 h_2/\varepsilon$  are the signed (i.e., they can be positive or negative) mesh Péclet numbers.

**Exercise 3.3.3.** Run cd2 with a homogeneous Neumann condition at the outflow boundary, and compare with the case in which the  $x_1$ -derivative at the outflow boundary is prescribed in accordance with the exact solution. This exercise illustrates once again that  $Pe \gg 1$  it is safe to prescribe a homogeneous Neumann condition at outflow.

Exercise 3.3.4. Run cd2 with the central scheme for the convection term. Observe that this does not give better results than the upwind scheme if the mesh Péclet number is larger than 2. The cause is the occurrence of (very small) wiggles. How small should  $dx_1$  be to bring the mesh Péclet number p below 2? Would p < 2 make the computing work nonuniform in  $\varepsilon$ ?

#### Some self-test questions

What is your favorite outflow boundary condition? Why?

Define the subcharacteristics of the convection-diffusion equation.

What is the difference between a regular and a singular perturbation problem? Formulate the matching principle.

What is the essential feature that makes it possible to have accuracy and efficiency Péclet-uniform? Why do we want this property?

When is a scheme of positive type? Why is this nice?

Under what conditions is the scheme with stencil of Exercise 3.3.2 of positive type?

# 4. The nonstationary convection-diffusion equation

# 4.1 Introduction

In the nonstationary case, time is included. The equation to be studied is the two-dimensional nonstationary convection-diffusion equation:

$$\frac{\partial \varphi}{\partial t} + (u_{\alpha}\varphi)_{,\alpha} - (\varepsilon\varphi_{,\alpha})_{,\alpha} = q(t, x_1, x_2) , 0 < t \le T, \quad (x_1, x_2) \in \Omega \equiv (0, 1) \times (0, 1) . \tag{4.1}$$

The following initial condition is required:

$$\varphi(0, x_1, x_2) = \varphi_0(x_1, x_2) . \tag{4.2}$$

Suitable boundary conditions are:

$$\varphi(t, x_1, x_2) = f(t, x_1, x_2) \text{ on } \partial \Omega_i \text{ (Dirichlet)},$$
 (4.3)

$$\varphi(t, x_1, x_2) = f(t, x_1, x_2)$$
 on  $\partial \Omega_o$  (Dirichlet) or (4.4)

$$\frac{\partial \varphi(t, x_1, x_2)}{\partial n} = g(t, x_1, x_2) \quad \text{on} \quad \partial \Omega_o \quad \text{(Neumann)}, \tag{4.5}$$

where **n** is the outward unit normal on the boundary  $\partial \Omega$ ,  $\partial \Omega_i$  is the inflow boundary (where  $u_{\alpha}n_{\alpha} < 0$ ) and  $\partial \Omega_o$  is the remainder of  $\partial \Omega$ , to be called the outflow boundary.

When  $\varepsilon = 1/\text{Pe} \ll 1$ , boundary layers may occur at the same location and with the same thickness as in the stationary case, as may be seen by means of singular perturbation theory, which is easily extended to the nonstationary case.

As in the stationary case, we have a maximum principle. The non-conservative form of (4.1) is:

$$\frac{\partial \varphi}{\partial t} + u_{\alpha} \varphi_{,\alpha} - (\varepsilon \varphi_{,\alpha})_{,\alpha} = \tilde{q} \equiv q - \varphi u_{\alpha}, \alpha . \tag{4.6}$$

The maximum principle for the nonstationary case says that if  $\tilde{q} \leq 0$  and if there is a local maximum in the interior of  $\Omega$  for  $t=t^*>0$ , then  $\varphi=$  constant,  $0 \leq t \leq t^*$ . In effect this says that local maxima can occur only at t=0 and at the boundaries. If  $\tilde{q} \geq 0$  the same applies to local minima, so that in the homogeneous case  $\tilde{q}=0$  there can be no local extrema (in space and time) in the interior; any numerical wiggles must be regarded as numerical artifacts. See Sect. 2.4 of Wesseling (2001) for a more precise version of the maximum principle. We will not prove this maximum principle here. But a simple physical analogy will convince the reader that it must be true. Think of a copper plate

with a nonuniform temperature distribution at t=0. The evolution of the temperature  $\varphi$  is governed by the heat equation, i.e. equation (4.6) with u=v=0. If no heat sources or sinks are present (i.e.  $\tilde{q}=0$ ) then the temperature distribution evolves to a uniform state, and a local hot spot must have been hotter at earlier times.

# Purpose of this chapter

The purpose of this chapter is:

- To introduce methods for discretization in time;
- To explain the concepts of consistency, stability and convergence and their relation:
- To show that numerical schemes must be stable to be convergent;
- To show how stability conditions can be derived by Fourier analysis for onestep schemes.
- How Fourier analysis can be used to derive stability conditions for general time-integration schemes by fitting the stability region.

# 4.2 A numerical example

Consider the one-dimensional heat equation, which is a special case of equation (4.1):

$$\frac{\partial \varphi}{\partial t} - \varepsilon \varphi_{,11} = 0 \; , \quad 0 < t \le T \; , \quad x \in \Omega \equiv (0,1) \; ,$$
 (4.7)

with initial condition and homogeneous Neumann boundary conditions given by

$$\varphi(0,x) = \varphi_0(x) , \quad \varphi_{,1}(t,0)\partial x = \varphi_{,1}(t,1)\partial x = 0 . \tag{4.8}$$

This is a mathematical model for the evolution of the temperature  $\varphi$  in a thin insulated bar with initial temperature distribution  $\varphi_0(x)$ . We have

$$\lim_{t \to \infty} \varphi(t, x) = \text{constant} = \int_0^1 \varphi_0(x) dx . \tag{4.9}$$

#### Discretization in space

For discretization in space we use the finite volume method on the vertexcentered grid of Fig. 2.1 with uniform mesh size h. The following details have been covered in the preceding chapters and we leave the derivation of (4.10) as an exercise. Integration over the control volumes gives, with  $\varepsilon$  constant:

$$\frac{h}{2} \frac{d\varphi_1}{dt} + F_1 - F_{3/2} = 0 ,$$

$$h \frac{d\varphi_j}{dt} + F_{j-1/2} - F_{j+1/2} = 0 , \quad j = 2, \dots, J - 1 ,$$

$$\frac{h}{2} \frac{d\varphi_J}{dt} + F_{J-1/2} - F_J = 0 ,$$
(4.10)

where F is an approximation of  $\varepsilon \partial \varphi / \partial x$ , naturally chosen as follows:

$$F_{j+1/2} = \varepsilon(\varphi_{j+1} - \varphi_j)/h$$
,  $j = 1, \dots, J-1$ .

From the boundary conditions it follows that  $F_1 = F_J = 0$ .

#### Discretization in time

Equation (4.10) is rewritten as

$$\frac{d\varphi_j}{dt} + L_h \varphi_j = 0 , \quad j = 1, \cdots, J.$$
 (4.11)

For discretization in time we choose the forward Euler scheme:

$$(\varphi_i^{n+1} - \varphi_j^n)/\tau + L_h \varphi_j^n = 0 , \quad n = 0, \dots, N , \quad N \equiv T/\tau ,$$
 (4.12)

with  $\tau$  the time step, taken constant;  $\varphi_j^n$  is the numerical approximation of  $\varphi(n\tau, x_j)$  with  $\varphi(t, x)$  the exact solution. From Fig. 2.1 it follows that  $x_j = (j-1)h, \ h = 1/(J-1)$ . Equation (4.12) is equivalent to

$$\varphi_1^{n+1} = (1-2d)\varphi_1^n + 2d\varphi_2^n , 
\varphi_j^{n+1} = d\varphi_{j-1}^n + (1-2d)\varphi_j^n + d\varphi_{j+1}^n , \quad j = 2, \dots, J-1 , 
\varphi_I^{n+1} = 2d\varphi_{I-1}^n + (1-2d)\varphi_I^n ,$$
(4.13)

where  $d \equiv \varepsilon \tau/h^2$  is a dimensionless number, that we will call the *diffusion* number. Note that this scheme is an approximation of  $\tau$  times the differential equation.

#### The matrix of the scheme

If we define  $\varphi \equiv (\varphi_1, \dots, \varphi_J)^T$  then (4.13) can be rewritten as

$$\varphi^{n+1} = A\varphi^n \; ,$$

with A the following tridiagonal matrix:

$$A = \begin{bmatrix} 1 - 2d & 2d & 0 & \cdots & 0 \\ d & 1 - 2d & d & & \vdots \\ 0 & \ddots & \ddots & \ddots & 0 \\ \vdots & & d & 1 - 2d & d \\ 0 & \cdots & 0 & 2d & 1 - 2d \end{bmatrix} . \tag{4.14}$$

#### Numerical results

The following numerical results have been obtained with the MATLAB code heq . As initial solution we choose

$$\varphi_0(x) = 0, \quad 0 < x < 0.4, 
\varphi_0(x) = 1, \quad 0.4 < x < 0.6, 
\varphi_0(x) = 0, \quad 0.6 < x < 1.$$
(4.15)

Numerical results are shown in Fig. 4.1 for two values of the diffusion number d. The result at the left looks as expected: the temperature distribution is tending

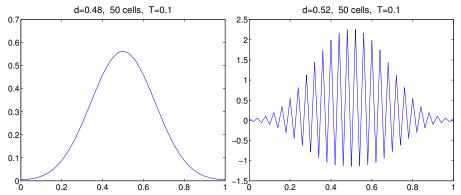


Figure 4.1. Numerical solution of heat equation for two values of d.

to uniformity, and the maximum principle is satisfied: no new maxima in space, and the maximum at x=1/2 was larger at earlier times. But the result in the right part of the figure is wrong. Clearly, the value of the diffusion number has a crucial influence, and something strange happens between d=0.48 and d=0.52! This will be investigated in what follows.

# Exercise 4.2.1. Prove equation (4.9).

Hints: Show that if  $\frac{\partial \varphi}{\partial t} = 0$  the solution is constant. Show that  $\frac{d}{dt} \int_0^1 \varphi dx = 0$ . (This shows that with these boundary conditions the bar is insulated: the heat content is conserved).

Exercise 4.2.2. Derive equations (4.10).

# 4.3 Convergence, consistency and stability

#### One-step schemes

Schemes in which only two time levels are involved are called *one-step schemes*; the forward Euler method (4.12) is an example. The general form of linear one-step schemes is:

$$B_1 \varphi^{n+1} = B_0 \varphi^n + B_2 \mathbf{q}^n + B_3 \mathbf{q}^{n+1} , \qquad (4.16)$$

where  $B_0, \dots, B_3$  are linear operators, and where  $\mathbf{q}$  arises from the right-hand side of the differential equation and the boundary conditions. Initially, we will restrict ourselves here to one-step schemes. Note that the 4 operators are the result of two different weighting operators.

#### Local and global truncation error

Let us denote the algebraic vector with elements equal to the exact solution evaluated at the grid points and at time  $t_n$  by  $\varphi_e^n$ . Let us define the *local truncation error*  $\tau_j^n$ , gathered in an algebraic vector  $\boldsymbol{\tau}^n$  (not to be confused with the time step  $\tau$ ) by the following equation:

$$B_1 \varphi_e^{n+1} = B_0 \varphi_e^n + B_2 \mathbf{q}^n + B_3 \mathbf{q}^{n+1} + \boldsymbol{\tau}^n . \tag{4.17}$$

The global truncation error is defined as

$$\mathbf{e}^n \equiv \boldsymbol{\varphi}_e^n - \boldsymbol{\varphi}^n \,. \tag{4.18}$$

By subtracting (4.17) and (4.16) we get the following relation between the global and the local truncation error:

$$B_1 \mathbf{e}^{n+1} = B_0 \mathbf{e}^n + \boldsymbol{\tau}^n \ . \tag{4.19}$$

This is very similar to equation (2.58), so that the above definitions are consistent with the truncation error definitions for the stationary case given in Sect. 2.3.

## Convergence

Of course, we want  $\|\mathbf{e}^n\| \downarrow 0$ ,  $n = 1, \dots, T/\tau$  as the grid is refined and  $\tau \downarrow 0$  (while keeping  $T/\tau$  integer, obviously). This is not the case for our previous numerical example when d > 1/2. No matter how small h and  $\tau$  are chosen, the numerical solution will always look like the right part of Fig. 4.1, since it depends only on the parameter d, cf. (4.13).

Clearly, we have to do some analysis to find conditions that guarantee that a numerical scheme is good. The fact that we made an  $\mathcal{O}(h^2)$  approximation of the spatial derivative and a  $\mathcal{O}(h)$  approximation to the temporal derivative is not enough to have the solution approach the exact solution.

Let the number of space dimensions be m and let us have an m-dimensional spatial grid G with grid points

$$\mathbf{x}_j = (x_{j_1}^1, \cdots, x_{j_m}^m) .$$

Hence, now j is a multi-index  $(j_1, \dots, j_m)$  (Fig. 4.2). Let the spatial mesh sizes and the time step be decreasing functions of a parameter h, that belongs to a sequence that decreases to zero. To emphasize the dependence of G on h we write  $G_h$ . We want that at a fixed time T and a fixed location  $\mathbf{x}$  in space the error to tend to 0 as  $h \downarrow 0$ . This implies that the number of time steps  $n = T/\tau$  changes, and the multi-index j changes to keep  $\mathbf{x}_j$  fixed. To emphasize this we write  $j_h$ . Of course, we assume that  $G_h$  is such that the fixed point  $\mathbf{x}_{j_h}$  remains a grid point when the grid is refined. For example, in the preceding numerical example we could choose

$$\mathbf{x}_{i_h} = 1/2$$
,  $h \in \{1/2, 1/4, 1/6, \cdots\}$ .

so that  $j_h \in \{1, 2, 3, \dots\}$ . We are now ready for the definition of *convergence*.

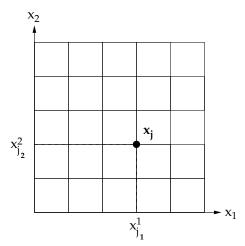


Figure 4.2. Example of use of the multi-index for m=2.

# **Definition 4.3.1.** Convergence

A scheme is called *convergent* if the global truncation error satisfies

$$\lim_{h\downarrow 0} e_{j_h}^{T/\tau} = 0, \quad \boldsymbol{x}_{j_h} \text{ fixed.}$$

Clearly, scheme (4.13) is not convergent for d > 1/2.

# Consistency

It seems likely that for convergence it is necessary that the local truncation error is small enough. We therefore formulate a condition. Let the scheme (4.16) in a given grid point  $\mathbf{x}_j$  approximate the differential equation times  $\tau^{\alpha}|\partial\Omega_j|^{\beta}$  with  $|\partial\Omega_j|$  the volume of some cell around  $\mathbf{x}_j$  (For example, the scheme (4.13) obviously approximates equation (4.7) times  $\tau$ , so that in this case  $\alpha=1,\ \beta=0$ ). This is necessary because the local truncation error changes when the whole scheme is premultiplied by  $\tau^{\alpha}|\partial\Omega_j|^{\beta}$ . We define

# **Definition 4.3.2.** Consistency

The scheme (4.16) is called *consistent* if

$$\lim_{h\downarrow 0} \tau_{j_h}^n/(\tau^\alpha |\partial \Omega_{j_h}|^\beta) = 0, \quad j \in G_h, \quad 1 \le n \le T/\tau .$$

In Exercise 4.3.1 the reader is asked to show that scheme (2.60) is *not* consistent on rough grids. This seems disturbing, because we are going to show that consistency is necessary for convergence, but in Sect. 2.3 it was shown that scheme (2.60) converges on rough grids. This apparent paradox arises from the fact that Def. 4.3.2 implies that the local truncation error is measured in the maximum norm. On rough grids we have consistency of scheme (2.60) in a more sophisticated norm, but not in the maximum norm; we will not go into this further. In this chapter only uniform grids are considered; on these grids the various appropriate norms are the same.

#### Stability

As illustrated by the numerical example in the preceding section, consistency does not imply convergence. It turns out the scheme requires another property to be convergent. In addition, *stability* is required. This concept will now be explained. Let  $\delta^0$  be a hypothetical arbitrary perturbation of  $\varphi^0$ . The resulting perturbation of  $\varphi^n$  is called  $\delta^n$ . It is left for the reader to derive from equation (4.16) that

$$B_1 \boldsymbol{\delta}^{n+1} = B_0 \boldsymbol{\delta}^n \ . \tag{4.20}$$

Let  $\|\cdot\|_h$  be some norm for functions  $G_h \to \mathbb{R}$ . Stability means that  $\delta^n$  remains bounded as  $n \to \infty$ , for all  $\delta^0$ . Two useful definitions are:

#### **Definition 4.3.3.** Zero-stability

A scheme is called *zero-stable* if there exists a bounded function C(T) and a function  $\tau_0(h)$  such that for arbitrary  $\delta^0$ 

$$\|\boldsymbol{\delta}^{T/\tau}\|_h \le C(T)\|\boldsymbol{\delta}^0\|_h \tag{4.21}$$

for all  $\tau \leq \tau_0(h)$  and all  $h \leq h_0$  for some fixed  $h_0$ .

The appellation "zero-stability" refers to the fact that the limit  $h \downarrow 0$  is considered. Note that  $\delta$  is allowed to grow slowly in each time step as long as the grow factor in each time step scales appropriately with  $\tau$ .

#### **Definition 4.3.4.** Absolute stability

A scheme is called *absolutely stable* if there exists a constant C and a function  $\tau_0(h)$  such that for arbitrary  $\boldsymbol{\delta}^0$ 

$$\|\boldsymbol{\delta}^n\|_h \le C\|\boldsymbol{\delta}^0\|_h \tag{4.22}$$

for h fixed, all n > 0 and all  $\tau \le \tau_0(h)$ .

The difference with zero-stability is that here h is fixed;  $\|\boldsymbol{\delta}^n\|_h$  can not grow in a time step, but any growth would jeopardize the bound on  $\lim_{n\to\infty}\|\boldsymbol{\delta}^n\|_h$ 

#### Lax's equivalence theorem

Definition 4.3.3 considers the perturbation at a fixed time T as  $h \downarrow 0$ , which is the same limit as in the definition of convergence. It can be shown that convergence implies zero-stability, and zero-stability plus consistency imply convergence. This is known as Lax's equivalence theorem. As a consequence, zero-stability is necessary for convergence. But absolute stability is also good to have, because it allows n to grow indefinitely, making it possible to continue time stepping until a steady state is reached. Absolute and zero-stability are not completely equivalent.

### A remark on stability analysis

The purpose of stability analysis is to find a suitable function  $\tau_0(h)$  such that (4.21) and/or (4.22) hold(s). This is the case if the linear operators in (4.20) satisfy

$$||(B_1^{-1}B_0)^n||_h \leq C$$
.

In the case of absolute stability, h and hence the dimensions of the matrices  $B_0$  and  $B_1$  are fixed, and linear algebra can be used to find conditions under which  $\|(B_1^{-1}B_0)^n\|$  is bounded as  $n \to \infty$ . But in the case of zero-stability,  $n \to \infty$  and  $h \downarrow 0$  simultaneously, and we have to study not just the behavior of the nth power of a matrix, but of a family of matrices of increasing size. This is not a familiar situation in linear algebra. We will see that Fourier analysis is well-suited to the study of both kinds of stability, if the boundary conditions are periodic.

**Exercise 4.3.1.** Show that for scheme (2.60) we have  $\beta = 1$ . Because this scheme is for the stationary case, time can be disregarded. Show that, using (2.66), scheme (2.60) is consistent on smooth grids (as defined below equation (2.66)), but *not* on rough grids.

# 4.4 Fourier stability analysis

## Applicability of Fourier analysis

In general it is difficult to derive estimates like (4.21) and (4.22). But if the coefficients in the scheme are constant, the mesh uniform, the grid a rectangular block and the boundary conditions periodic, then Fourier analysis applies and the required estimates are often not difficult to obtain.

In practice, of course, the coefficients are usually not constant. The scheme is called *locally stable* if we have stability for the constant coefficients scheme that results from taking local values of the coefficients, and to assign these values to the coefficients in the whole domain (frozen coefficients method). Local stability in the whole domain is necessary for stability in the variable coefficients case. We will discuss stability only for constant coefficients.

Stability theory for non-periodic boundary conditions is complicated. But for explicit time stepping schemes it takes a while before the influence of the boundary conditions makes itself felt in the interior, so that Fourier stability theory applies during a certain initial time span. As a consequence, stability with periodic boundary conditions is desirable, even if the boundary conditions are of different type.

## Example of frozen coefficients method

Consider the Burgers equation:

$$\frac{\partial \varphi}{\partial t} + \frac{1}{2} \left( \varphi^2 \right)_{,1} = 0.$$

Discretization with the forward Euler method in time and the upwind scheme in space gives:

$$\varphi_j^{n+1} - \varphi_j^n + \frac{\tau}{2h} [(\varphi_j^n)^2 - (\varphi_{j-1}^n)^2] = 0,$$

assuming  $\varphi>0$  and a uniform grid. For stability analysis we postulate a perturbation  $\delta\varphi^0$  of the initial solution. The perturbed solution satisfies

$$(\varphi + \delta\varphi)_j^{n+1} - (\varphi + \delta\varphi)_j^n + \frac{\tau}{2h} \{ [(\varphi + \delta\varphi)_j^n]^2 - [(\varphi + \delta\varphi)_{j-1}^n]^2 \} = 0.$$

Subtraction of the preceding two equations and linearization (i.e. deletion of terms quadratic in  $\delta\varphi$ ) gives:

$$\delta\varphi_j^{n+1} - \delta\varphi_j^n + \frac{\tau}{h}[(\varphi\delta\varphi)_j^n - (\varphi\delta\varphi)_{j-1}^n] = 0.$$

Freezing of the coefficients results in

$$\delta\varphi_j^{n+1} - \delta\varphi_j^n + \frac{c\tau}{h}(\delta\varphi_j^n - \delta\varphi_{j-1}^n) = 0,$$

where c is the frozen value of  $\varphi$ . This scheme allows Fourier stability analysis. Usually, a stability condition of the type

$$\frac{c\tau}{h} < C \tag{4.23}$$

results for some value of C. This condition is to be satisfied for the frozen coefficient c equal to all values that the variable coefficient  $\varphi_j^n$  takes. We see from (4.23) that it suffices to take  $c = \max(|\varphi_j^n|)$ . Frequently an informed guess for  $\max(|\varphi_j^n|)$  can be made by looking at the boundary conditions.

In the remainder of this section we present the basic principles of Fourier stability analysis.

### Fourier series

Let  $G_h$  be a uniform grid on the unit interval with nodes

$$x_i = jh$$
,  $j = 0, 1, \dots, J - 1 \equiv 1/h$ .

It can be shown that every grid function  $\delta: G_h \to \mathbb{R}$  can be represented by what is called a *Fourier series* Wesseling (1992):

$$\delta_j = \sum_{k=-m}^{m+p} c_k e^{ij2\pi k/J}, \quad j = 0, 1, \dots, J-1,$$
 (4.24)

where p = 0, m = (J - 1)/2 for J odd and p = 1, m = (J - 2)/2 for J even. In turn the coefficients  $c_k$  of the expansion are given as:

$$c_k = J^{-1} \sum_{j=0}^{J-1} \delta_j e^{-ij2\pi k/J}$$
(4.25)

We first need to show that the different discrete Fourier modes are orthogonal:

$$\sum_{i=0}^{J-1} e^{ij2\pi k/J} e^{-ij2\pi l/J} = J\delta_{kl}$$
 (4.26)

When l = k we have

$$\sum_{j=0}^{J-1} e^{ij2\pi k/J} e^{-ij2\pi k/J} = \sum_{j=0}^{J-1} e^0 = J$$
 (4.27)

when  $l \neq k$  we have a geometric series. Recall that

$$s_n = \sum_{j=1}^n ar^{j-1} = \frac{a(1-r^n)}{1-r}$$
(4.28)

in our case:

$$\sum_{j=0}^{J-1} e^{ij2\pi k/J} e^{-ij2\pi l/J} = \sum_{j=0}^{J-1} (e^{i2\pi(k-l)/J})^j =$$

$$\frac{1 - e^{iJ2\pi(k-l)/J}}{1 - e^{i2\pi(k-l)/J}} = 0$$

$$(4.29)$$

We can now prove (4.24) given that (4.25), holds:

$$\sum_{k=-m}^{m+p} c_k e^{ij2\pi k/J} = J^{-1} \sum_{k=-m}^{m+p} \sum_{l=0}^{J-1} \delta_l e^{-il2\pi k/J} e^{ij2\pi k/J} =$$

$$J^{-1} \sum_{l=0}^{J-1} \delta_l \sum_{k=0}^{J-1} e^{i2\pi (k-m)(j-l)/J} =$$

$$J^{-1} \sum_{l=0}^{J-1} \delta_l e^{i2\pi m(l-j)/J} \sum_{k=0}^{J-1} e^{ik2\pi jk/J} e^{-ik2\pi lk/J}$$

$$(4.30)$$

Next we follow the reverse course and show that given (4.25) we can prove (4.24):

$$J^{-1} \sum_{j=0}^{J-1} \delta_j e^{-ij2\pi k/J} = J^{-1} \sum_{l=-m}^{l=m+p} c_l \sum_{j=0}^{J-1} e^{-ij2\pi k/J} e^{ij2\pi l/J} =$$

$$\sum_{l=-m}^{l=m+p} c_l \delta_{kl} = c_k$$

$$(4.31)$$

It is convenient to rewrite this as

$$\delta_j = \sum_{\theta \in \Theta} c_{\theta} e^{ij\theta} , \quad \Theta \equiv \{\theta = 2\pi k/J , k = -m, -m+1, \cdots, m+p\} . \quad (4.32)$$

We note that  $e^{ij\theta} = \cos j\theta + i\sin j\theta$  is complex, so that  $c_{\theta}$  is also complex, such that  $\delta_j$  is real. We can regard  $c_{\theta}$  as the amplitude of the harmonic wave  $e^{ij\theta}$ .

In (4.32)  $\theta$  ranges approximately between  $-\pi$  and  $\pi$  if  $J \gg 1$ . For  $\theta = \pi$  we have the shortest wave that can be resolved on  $G_h$ :  $e^{ij\pi} = (-1)^j$ , and for  $\theta = 0$  the wavelength is infinite:  $e^0 = 1$ . Note that  $\boldsymbol{\delta}$  is periodic:  $\delta_j = \delta_{j+J}$ . The functions  $e^{ij\theta}$ ,  $\theta \in \Theta$  are called Fourier modes. The parameter  $\theta$  is called the wavenumber. The function  $f(x) \equiv e^{ij\theta}$  has period or wavelength  $2\pi/\theta$ , since  $f(x + 2\pi/\theta) = f(x)$ .

The Fourier series (4.32) is easily extended to more dimensions. We restrict ourselves to the two-dimensional case. Let  $G_h$  be a uniform grid on the unit square with nodes

$$x_i = (j_1 h_1, j_2 h_2), \quad j_{\alpha} = 0, \dots, J_{\alpha} - 1 \equiv 1/h_{\alpha}, \quad \alpha = 1, 2.$$

Define the set  $\Theta$  of wavenumbers as

$$\Theta \equiv \{ \theta = (\theta_1, \ \theta_2) : \quad \theta_{\alpha} = 2\pi k_{\alpha}/J_{\alpha} ,$$

$$k_{\alpha} = -m_{\alpha}, -m_{\alpha} + 1, \cdots, m_{\alpha} + p_{\alpha} , \quad \alpha = 1, 2 \} ,$$

where  $p_{\alpha}=0,\ m_{\alpha}=(J_{\alpha}-1)/2$  for  $J_{\alpha}$  odd and  $p_{\alpha}=1,\ m_{\alpha}=J_{\alpha}/2-1$  for  $J_{\alpha}$  even. Define

$$j\theta = \sum_{\alpha=1}^{2} j_{\alpha}\theta_{\alpha} . \tag{4.33}$$

It can be shown that every grid function  $\delta: G_h \to \mathbb{R}$  can be written as

$$\delta_j = \sum_{\theta \in \Theta} c_\theta e^{ij\theta} \ . \tag{4.34}$$

with the amplitudes  $c_{\theta}$  given by

$$c_{\theta} = N^{-1} \sum_{j \in G_h} \delta_j e^{-ij\theta} , \quad N = J_1 J_2 .$$

As a consequence we have

$$\sum_{\theta \in \Theta} c_{\theta} e^{ij\theta} = 0, \quad \forall j \in G_h \qquad \Rightarrow \qquad c_{\theta} = 0, \quad \forall \theta \in \Theta . \tag{4.35}$$

Let us define the  $l_2$ -norm by

$$\|\boldsymbol{\delta}\| = N^{-\frac{1}{2}} \{ \sum_{j \in G_h} (\delta_j)^2 \}^{1/2}, \quad \|\boldsymbol{c}\| = N^{-\frac{1}{2}} \{ \sum_{\theta \in \Theta} |c_{\theta}|^2 \}^{1/2}$$

We will need

Theorem 4.4.1. Parseval.

If  $\delta$  and c are related by (4.34), then in the  $l_2$ -norm

$$\|\boldsymbol{\delta}\| = N^{-1/2}\|\boldsymbol{c}\|$$
.

### An example

Consider the example of Sect. 4.2. The perturbation  $\boldsymbol{\delta}^n$  in the numerical solution  $\boldsymbol{\varphi}^n$  satisfies the same equation as  $\boldsymbol{\varphi}^n$ , i.e. equation (4.13) (show this!). But now we assume periodic boundary conditions (to make Fourier analysis applicable), so that the boundary conditions are replaced by the condition  $\delta_j = \delta_{j+J}$ . We have

$$\delta_{i}^{n+1} = d\delta_{i-1}^{n} + (1 - 2d)\delta_{i}^{n} + d\delta_{i+1}^{n}.$$

Substitution of (4.32) gives:

$$\sum_{\theta \in \Theta} e^{ij\theta} \left[ c_{\theta}^{n+1} - c_{\theta}^{n} (de^{-i\theta} + 1 - 2d + de^{i\theta}) \right] = 0.$$

Because  $\varepsilon$  and h are constant, d is constant; therefore the term between [ ] does not depend on j; if this were not the case the following step could not be taken and Fourier analysis falls through. But since the term between [ ] does not depend on j, it follows from (4.35) that the term between [ ] is zero, and we get rid of the sum in the preceding equation. Using  $e^{-i\theta} + e^{i\theta} = 2\cos\theta$  we get

$$c_{\theta}^{n+1} = g(\theta)c_{\theta}^{n} , \quad g(\theta) \equiv 1 - 2d(1 - \cos\theta) , \qquad (4.36)$$

where  $g(\theta)$  is called the *amplification factor*: it measures the amplification (or damping) of the amplitude  $c_{\theta}$  of the Fourier mode  $e^{ij\theta}$ . Before continuing with this example, we go to the general case.

# The general case

Whenever Fourier stability analysis is applicable, we get relation (4.36) with some function  $g(\theta)$  (that is complex in general). It follows that

$$c_{\theta}^{n} = g(\theta)^{n} c_{\theta}^{0} ,$$

so that

$$\|\boldsymbol{c}^n\| \le \bar{g}^n \|\boldsymbol{c}^0\|, \quad \bar{g} = \max\{|g(\theta)|: \theta \in \Theta\},$$
 (4.37)

with equality for the  $\theta$  for which the maximum is attained. According to Parseval's theorem (Theorem 4.4.1) we have  $\|\boldsymbol{c}\| = N^{-1/2} \|\boldsymbol{\delta}\|$ , so that the preceding equation gives

$$\|\boldsymbol{\delta}^n\| < \bar{q}^n \|\boldsymbol{\delta}^0\| . \tag{4.38}$$

From Definition 4.3.4 it follows that for absolute stability we must have

$$\bar{g}^n < C, \quad \forall n$$

for some C; hence

$$\bar{q} < 1. \tag{4.39}$$

This is sufficient, but also necessary, because equality can occur in (4.37), since all Fourier modes can be present, because  $\delta^0$  is arbitrary (cf. Def. 4.3.4).

A sufficient condition for zero-stability is that there exists a constant C such that

$$\bar{g}^{(T/\tau)} \le C \tag{4.40}$$

for  $0 \le \tau \le \tau_0(h)$ . Since

$$C^{\tau/T} = \exp(\frac{\tau}{T} \ln C) = 1 + \mathcal{O}(\tau) , \qquad (4.41)$$

we may write

$$\bar{g} \le 1 + \mathcal{O}(\tau) \ . \tag{4.42}$$

This is the von Neumann condition for zero-stability (after John von Neumann, who introduced the Fourier method for stability analysis around 1944 in Los Alamos; not to be confused with the nineteenth century mathematician Neumann of the boundary condition). The von Neumann condition is also necessary, because if  $\bar{g} \geq 1 + \mu$ ,  $\mu > 0$ , then there is a  $\theta$  with  $|g(\theta)| = 1 + \mu$ . Choosing  $\delta_j^0 = e^{ij\theta}$  gives  $\|\boldsymbol{\delta}^n\|/\|\boldsymbol{\delta}\| = (1 + \mu)^n$ ,  $n = T/\tau$ , which is unbounded as  $\tau \downarrow 0$ . Note that the  $\mathcal{O}(\tau)$  term (4.42) is not allowed to depend on h; this follows from (4.40)—(4.42).

We will neglect the  $\mathcal{O}(\tau)$  term in (4.42), because this makes hardly any difference. For simplicity, we will also extend the set of wave numbers  $\Theta$  to

$$\Theta = (-\pi, \ \pi]$$
 (one dimension),  $\Theta = (-\pi, \ \pi] \times (-\pi, \ \pi]$  (two dimensions).

This also makes hardly any difference, since in practice  $J_{\alpha} \gg 1$ . We end up with the following condition for absolute and zero-stability:

$$\bar{g} \le 1 \;, \quad \bar{g} \equiv \max\{|g(\theta)| : \forall \theta\}.$$
 (4.43)

(Note that  $g(\theta)$  is complex in general, so that |g| is the modulus of a complex number).

## An example, continued

In our example  $g(\theta)$  is given by (4.36), so that  $g(\theta)$  happens to be real, and equation (4.43) gives:

$$-1 < 1 - 2d(1 - \cos \theta) < 1$$
,  $\forall \theta$ .

Th right inequality is always satisfied, since d > 0. The left inequality gives

$$d(1-\cos\theta) \le 1$$
,  $\forall \theta$ ,

so that we must have

$$d \leq 1/2$$
.

This beautifully explains the numerical results obtained in Fig. 4.1. It follows that the time step must satisfy

$$\tau \le \frac{h^2}{2\varepsilon} \ . \tag{4.44}$$

Here we encounter an example of the function  $\tau_0(h)$  in the definition of zero-stability:

$$\tau_0(h) = \frac{h^2}{2\varepsilon} \ .$$

Condition (4.44) is rather restrictive, because  $\tau$  must be reduced much more than h, when h is decreased for better accuracy. It may be inefficient to use such small time steps. Therefore we will consider another scheme below. But first we give another example.

### Second example

Consider the one-dimensional convection equation:

$$\frac{\partial \varphi}{\partial t} + u\varphi_{,1} = 0. \tag{4.45}$$

We discretize in space with the upwind scheme and in time with the forward Euler scheme, and obtain (assuming u > 0):

$$\varphi_j^{n+1} = \varphi_j^n - c(\varphi_j^n - \varphi_{j-1}^n) , \quad c \equiv u\tau/h .$$
 (4.46)

The dimensionless number c is called the *Courant-Friedrichs-Lewy number* or *CFL number*, after the authors of the 1928 paper in which the importance of numerical stability was first brought to light; c is also called the *Courant number*. The quick way to get the amplification factor is to take just one Fourier mode, and to substitute  $\varphi_j^n = c_\theta^n e^{ij\theta}$ ; the amplification factor is  $g(\theta) = c_\theta^{n+1}/c_\theta^n$ . This gives

$$g(\theta) = 1 - c(1 - e^{-i\theta})$$
 (4.47)

To study stability we must consider  $|g(\theta)|$ , which, however, takes a somewhat unpleasant form, so that we prefer the following more elegant geometrical approach. We note that the complex number  $g(\theta)$ , when  $\theta$  varies between  $-\pi$  and  $\pi$ , traces out a circle with center at 1-c and radius c. From Fig. 4.4 it is clear that for  $|g(\theta)|$  not to leave the unit circle we must have  $0 \le c \le 1$ , resulting in the following stability condition on the time step:

$$\tau \le h/u \,, \tag{4.48}$$

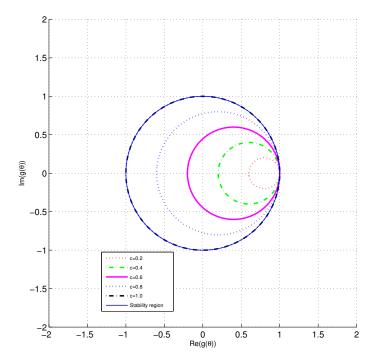
which provides another example of the function  $\tau_0(h)$  in Def. 4.3.3.

#### The $\omega$ -scheme

The backward Euler scheme for equation (4.11) is given by

$$(\varphi_j^{n+1} - \varphi_j^n)/\tau + L_h \varphi_j^{n+1} = 0.$$
 (4.49)

Now we have to solve a system of equations for  $\varphi^{n+1}$ , which is why this is called an *implicit* scheme; the forward Euler scheme (4.12) is of *explicit* type. Therefore a time step with an implicit scheme requires much more computing work than an explicit scheme, but this may be compensated by better stability properties, allowing a larger time step. The global truncation error of the Euler time stepping schemes (4.12) and (4.49) satisfies



**Figure 4.3.** Locus of  $g(\theta)$  in complex plane for different values of parameter c in (4.47).

$$\mathbf{e} = \mathcal{O}(\tau + h^m) ,$$

where m depends on the accuracy of the spatial discretization  $L_h$ . We can improve this by taking a linear combination of the forward and backward Euler schemes, as follows:

$$(\varphi_{jk}^{n+1} - \varphi_{jk}^{n})/\tau + (1 - \omega)L_h\varphi_{jk}^{n} + \omega L_h\varphi_{jk}^{n+1} = \omega q_{jk}^{n+1} + (1 - \omega)q_{jk}^{n}, \quad (4.50)$$

where we now assume the right-hand side q in (4.1) to be nonzero, and where we have gone to the two-dimensional case. This is called the  $\omega$ -scheme. For  $\omega = 1/2$  this is called the *Crank-Nicolson scheme*, which is second order in time:

$$\mathbf{e} = \mathcal{O}(\tau^2 + h^m) \ .$$

With the central scheme for convection, the space discretization of equation (4.1) is, with  $u, v, \varepsilon$  constant and uniform mesh sizes  $h_1$  and  $h_2$  in the x- and y-directions, respectively,:

$$L_{h}\varphi_{jk} = \frac{u}{2h_{1}}(\varphi_{j+1,k} - \varphi_{j-1,k}) + \frac{v}{2h_{2}}(\varphi_{j,k+1} - \varphi_{j,k-1}) + \frac{\varepsilon}{h_{1}^{2}}(-\varphi_{j-1,k} + 2\varphi_{jk} - \varphi_{j+1,k}) + \frac{\varepsilon}{h_{2}^{2}}(-\varphi_{j,k-1} + 2\varphi_{jk} - \varphi_{j,k+1}).$$

$$(4.51)$$

The equation for the perturbation  $\delta^n_{jk}$  is identical to the homogeneous version  $(i.e.\ q=0)$  of equation (4.51). For  $1/2<\omega<1$  the stability analysis of the  $\omega$ -scheme is easy. As said before, the quick way to determine the amplification factor is to substitute  $\delta^n_j=c^n_\theta e^{ij\theta}$ , where now j stands for  $\{j,k\}$ , and where  $j\theta=j\theta_1+k\theta_2,\ cf.$  equation (4.33). Substitution gives

$$c_{\theta}^{n+1} - c_{\theta}^{n} + (1 - \omega)\tau \hat{L}_{h}(\theta)c_{\theta}^{n} + \omega\tau \hat{L}_{h}(\theta)c_{\theta}^{n+1} = 0,$$

where

$$\hat{L}_h(\theta) \equiv e^{-ij\theta} L_h e^{ij\theta} \,. \tag{4.52}$$

commonly referred to as the Fourier transform, or the symbol, of  $L_h$ . It follows that

$$g(\theta) = \frac{c_{\theta}^{n+1}}{c_{\theta}^{n}} = \frac{1 - (1 - \omega)\tau \hat{L}_{h}(\theta)}{1 + \omega\tau \hat{L}_{h}(\theta)}.$$

Let the real and imaginary part of the complex variable  $\tau \hat{L}_h$  be  $w_1$  and  $w_2$ , respectively:  $\tau \hat{L}_h = w_1 + iw_2$ . Noting that for two complex numbers  $z_{1,2}$  we have  $|z_1/z_2| = |z_1|/|z_2|$  and noting that  $|\hat{L}_h|^2 = w_1^2 + w_2^2$ , we get

$$|g(\theta)|^2 \equiv \frac{[1 - (1 - \omega)w_1]^2 + (1 - \omega)^2 w_2^2}{(1 + \omega w_1)^2 + \omega^2 w_2^2} .$$

Assume that  $w_1 \geq 0$ . Then for  $1/2 \leq \omega \leq 1$  the denominator is not smaller than the numerator, so that  $|g(\theta)| \leq 1$ , and we have stability. To check whether  $w_1 \geq 0$ , we have to determine  $\tau \hat{L}_h$ . We find that

$$\tau \hat{L}_h = i(c_1 \sin \theta_1 + c_2 \sin \theta_2) + 2d_1(1 - \cos \theta_1) + 2d_2(1 - \cos \theta_2) , \qquad (4.53)$$

where

$$c_1 \equiv \frac{u\tau}{h_1} \; , \; c_2 \equiv \frac{v\tau}{h_2} \; , \; d_1 \equiv \frac{\varepsilon\tau}{h_1^2} \; , \; d_2 \equiv \frac{\varepsilon\tau}{h_2^2} \; ,$$

from which it is obvious that  $w_1 = \text{Re}(\tau \hat{L}_h) \geq 0$ ,  $\forall \theta$ . Similarly, with the upwind scheme for convection we obtain, assuming  $u, v \geq 0$ ,

$$\tau \hat{L}_h = c_1 (1 - e^{-i\theta_1}) + c_2 (1 - e^{-i\theta_2}) + 2d_1 (1 - \cos \theta_1) + 2d_2 (1 - \cos \theta_2) , \quad (4.54)$$

We have 
$$w_1 = (c_1 + 2d_1)(1 - \cos \theta_1) + (c_2 + 2d_2)(1 - \cos \theta_2) \ge 0$$
,  $\forall \theta$ , since  $c_{1,2} \ge 0$ .

Hence, we have established unconditional stability of the  $\omega$ -scheme for the convection-diffusion equation, for  $1/2 \leq \omega \leq 1$ . The only interesting values of  $\omega$  are 0,  $1/2 + \mathcal{O}(\tau)$ , 1. The value  $\omega = 0$  is of interest because this gives an explicit scheme, for which a time step is cheap. A value  $\omega = 1/2 + \mathcal{O}(\tau)$  is of interest because this gives  $\mathcal{O}(\tau^2)$  accuracy. Finally,  $\omega = 1$  is of interest because this is necessary for the discrete maximum principle in the nonstationary case; we will not go into this. Therefore we will not give stability conditions for the  $\omega$ -scheme for  $0 \leq \omega < 1/2$ , but only for  $\omega = 0$ .

In this case the analysis proceeds as follows (Wesseling (2001)). Using the Fourier transform of the solution and (4.52) we can write (4.1) as

$$\frac{dc_{\theta}}{dt} = -\hat{L}_h(\theta)c_{\theta}^n \tag{4.55}$$

The stability domain of numerical methods for the differential equation

$$\frac{dy}{dt} = \lambda y, \quad \lambda \in \mathbb{C},\tag{4.56}$$

is defined as

$$S = \left\{ \lambda \in \mathbb{C}; |y^{n+1}| \le |y^n| \right\} \tag{4.57}$$

Comparing (4.55) and (4.56) shows that for zero stability it is necessary and sufficient that  $-t\hat{L} - h(\theta) \in S$ ,  $\forall \theta \in \Theta$ . To simplify the analysis this requirement is strengthened to hold for all  $\theta$  (as opposed to the discrete values of  $\theta$  in  $\Theta$ . As a foundation for formulating stability conditions we require

$$S_L \subseteq S, \quad S_L \equiv \left\{ -\tau \hat{L}_h(\theta) : \forall \theta \right\}$$
 (4.58)

In words: For stability the symbol should be scaled with the time step to fit inside the stability region.

For the  $\kappa$ -scheme defined in Chapter 3, the symbol is given by

$$\tau \hat{L}_h(\theta) = \hat{C}_h(\theta) + \hat{D}_h(\theta), \tag{4.59}$$
$$\hat{C}_h(\theta) = \gamma_1(\theta) + i\gamma_2(\theta), \quad \hat{D}_h(\theta) = \delta(\theta),$$

where

$$\gamma_1(\theta) = 2(1 - \kappa) \sum_{\alpha = 1, 2} (c_{\alpha} s_{\alpha}^2) = 2 \sum_{\alpha = 1, 2} \tilde{c}_{\alpha} s_{\alpha}^2, \quad \tilde{c}_{\alpha} = (1 - \kappa) c_{\alpha}$$
(4.60)

$$\gamma_2(\theta) = \sum_{\alpha=1,2} \{(1-\kappa)s_\alpha + 1\} \sin \theta_\alpha, \quad s_\alpha = \sin^2 \frac{1}{2}\theta_\alpha$$
 (4.61)

$$\delta(\theta) = 2\sum_{\alpha=1,2} d_{\alpha} s_{\alpha} \tag{4.62}$$

 $c_{\alpha}$  and  $d_{\alpha}$  are the Courant-Friedrichs-Lewy (CFL) number and the diffusion number, respectively, defined as

$$c_{\alpha} = |u_{\alpha}|\tau/h_{\alpha}, \quad d_{\alpha} = 2\varepsilon\tau/h_{\alpha}^{2}$$
 (4.63)

In (Wesseling (2001)) a number of conditions can be derived on the coefficients of the discretised convection diffusion equation, i.e.  $\kappa$ ,  $c_{\alpha}$ ,  $d_{\alpha}$ , to have the symbol fit in the following different geometric shapes.

- Rectangle, Theorem (5.7.1) of (Wesseling (2001))
- Ellipse, Theorem (5.7.2) of (Wesseling (2001))
- Oval, Theorem (5.7.3) of (Wesseling (2001))
- Half ellipse, Theorem (5.7.4) of (Wesseling (2001))
- Parabola, Theorem (5.7.5) of (Wesseling (2001))

As an example we will use the conditions on the symbol derived in (Wesseling (2001)), to derive conditions on the time-step to have a stable time integration with the explicit Euler method and a central discretization for the convective terms. Any textbook on time-integration methods will show how to derive the stability region of this method: A unit circle in the complex plane, centered at (-1,0):

$$(v+1)^2 + w^2 = 1, \quad v + iw = z.$$
 (4.64)

We can now use Theorem (5.7.2) to define conditions on the coefficients in the discretised equation, such that the symbol will fit in this region.

Theorem 4.4.2. (Ellipse).

If

$$\tilde{d} = \sum_{\alpha=1,2} d_{\alpha} \le a \quad and \quad \sum_{\alpha=1,2} c_{\alpha}^{2} / d_{\alpha} \le (2 - \kappa)^{-2} b^{2} / a$$
 (4.65)

then  $S_L$  is contained in the ellipse

$$(v/a+1)^2 + (w/b)^2 = 1, \quad v+iw=z, \quad a>0.$$
 (4.66)

The first condition is necessary.

The proof is given in (Wesseling (2001)).

To fit in the stability region we have to choose a = b = 1. This means that the following restrictions on the time-step should hold.

$$d_1 + d_2 \le 1 \text{ and } \frac{c_1^2}{d_1} + \frac{c_2^2}{d_2} \le (2 - \kappa)^{-2}.$$
 (4.67)

For the central scheme  $\kappa = 1$  and the resulting conditions are

$$2\varepsilon\tau(\frac{1}{h_1^2} + \frac{1}{h_2^2}) \le 1 \text{ and } \frac{\tau u_\alpha u_\alpha}{2\varepsilon} \le 1.$$
 (4.68)

**Exercise 4.4.1.** Suppose we have a numerical scheme for the convection equation

$$\frac{\partial \varphi}{\partial t} + c \frac{\partial \varphi}{\partial r} = 0$$
,  $c \text{ constant}$ ,

that has a stability condition  $c\tau/h < C$ . What is the stability condition for this scheme for the nonlinear case  $\partial \varphi/\partial t + \partial \varphi^m/\partial x = 0$ ?

Exercise 4.4.2. Derive equations (4.46) and (4.47).

**Exercise 4.4.3.** Show that scheme (4.46) is unconditionally unstable when u < 0. (This is one way to see why the downwind scheme is bad).

**Exercise 4.4.4.** Discretize (4.45) with the central scheme in space and the forward Euler scheme in time. Show that the scheme is unconditionally unstable.

**Exercise 4.4.5.** This exercise is meant to demonstrate the attractiveness of the geometric approach illustrated in Fig. 4.4. Determine  $|g(\theta)|$ , with  $g(\theta)$  given by equation (4.47). Use analysis instead of geometry to find conditions on c such that  $|g(\theta)| \leq 1$ ,  $\forall \theta$ .

Exercise 4.4.6. Derive equations (4.51)—(4.54).

Exercise 4.4.7. Write down the upwind version of scheme (4.51).

# 4.5 Numerical experiments

#### Problem statement

Some numerical experiments will be presented for the following one-dimensional test problem:

$$\begin{split} \frac{\partial \varphi}{\partial t} + u\varphi_{,1} - \varepsilon \varphi_{,11} &= q, \qquad 0 < x < 1, \quad 0 < t \le T \;, \\ q(t,x) &= \beta^2 \varepsilon \cos \beta (x - ut) \;, \end{split} \tag{4.69}$$

with  $\varepsilon$  and u > 0 constant, and  $\beta$  a parameter. An exact solution is given by

$$\varphi(t,x) = \cos\beta(x - ut) + e^{-\alpha^2 \varepsilon t} \cos\alpha(x - ut) , \qquad (4.70)$$

with  $\alpha$  arbitrary. Spatial discretization is done with the second order central or with the first order upwind scheme on the vertex-centered grid of Fig. 2.1 with uniform mesh size h. For temporal discretization the  $\omega$ -scheme is used. The resulting scheme can be written as

$$\frac{h_j}{\tau}(\varphi_j^{n+1} - \varphi_j^n) + \omega L_h \varphi_j^{n+1} + (1 - \omega) L_h \varphi_j^n = h_j [\omega q_j^{n+1} + (1 - \omega) q_j^n], \quad (4.71)$$

where  $h_j$  is the volume of the cell over which is integrated. We choose a Neumann condition at x = 1, so that  $h_j = h$ ,  $j \neq J$ ,  $h_J = h/2$ . In the interior we get for the central scheme:

$$L_{h}\varphi_{j} = F_{j+1/2} - F_{j-1/2} ,$$

$$F_{j+1/2} = \frac{1}{2}u(\varphi_{j+1} + \varphi_{j}) - \frac{\varepsilon}{h}(\varphi_{j+1} - \varphi_{j}) , \quad j = 2, \dots, J-1 .$$
(4.72)

At the Neumann boundary we integrate over a half cell, and obtain

$$F_{J+1/2} = u\varphi_J - \varepsilon b(t) ,$$

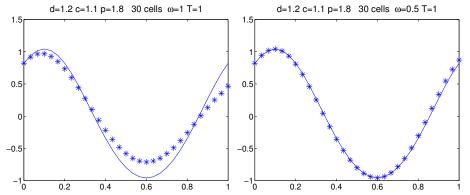
assuming a Neumann condition  $\partial \varphi(t,1)/\partial x = b(t)$ .

#### Choice of time step, mesh size and time scale

The length scale  $\mathcal{L}$  and time scale  $\mathcal{T}$  of the exact solution are given by

$$\mathcal{L} = \pi / \max(\alpha, \beta), \quad \mathcal{T} = \min\{\mathcal{L}/u, (\varepsilon \alpha^2)^{-1}\},$$

where we take for the length scale of a harmonic function half its wavelength. We may expect accuracy to be sufficient if  $\tau \ll \mathcal{T}$ ,  $h \ll \mathcal{L}$ . For efficiency, we would like to avoid more stringent restrictions on  $\tau$  and h, such as might arise from stability. In the numerical experiments to be described we take  $\alpha = 4\pi$ ,  $\beta = 2\pi$ , so that  $\mathcal{L} \cong 1/4$ . We take mostly h = 1/30, giving  $h/\mathcal{L} \cong 0.13$ .



**Figure 4.4.** Exact (—) and numerical solution (\*) of (4.69). Central scheme; u = 1.1,  $\varepsilon = 0.02$ ,  $h = \tau = 1/30$ , t = 1,  $\alpha = 4\pi$ ,  $\beta = 2\pi$ . Left:  $\omega = 1$ , right:  $\omega = 1/2$ .

#### Numerical results

The numerical experiments described below have been carried out with the code cdns. We prescribe a Dirichlet condition at x = 0 and a Neumann condition at x = 1. Initial and boundary conditions are chosen conforming with the exact solution (4.70). The left half of Fig. 4.4 shows a result. In this case we have  $\mathcal{T} \cong 0.23, \ \tau \cong 0.15\mathcal{T}$ . The accuracy with  $\omega = 1$  is disappointing. The cause is that the scheme is only first order accurate in time. With  $\omega = 1/2$  the scheme is second order accurate in time, and the accuracy is much better. In this case we have  $d \equiv 2\varepsilon \tau/h^2 = 1.2$ ,  $\tau u^2/(2\varepsilon) = 1.0$ , so that the explicit  $(\omega = 0)$  scheme is instable according to the one-dimensional version  $(v=0, h_2=\infty)$  of the stability conditions To get comparable accuracy as in the right part of the figure with  $\omega = 0$  we find that we need to decrease (in order to compensate for first order accuracy in time)  $\tau$  to 1/300 (results not shown). This gives an elapsed time (with tic...toc) of 0.287, whereas for the right part of the figure we find an elapsed time of 0.027. The stability of the explicit ( $\omega = 0$ ) scheme constrains the time step much more than accuracy. This is inefficient, so that it pays of to use a more complicated scheme with more work per time step, but with a less severe stability restriction on the time step.

Exercise 4.5.1. Verify the stability conditions by numerical experiments with the MATLAB program used in this section.

Exercise 4.5.2. Make a cell-centered version of the code cdns .

#### Some self-test questions

Write down the instationary convection-diffusion equation.

Formulate the maximum principle for the instationary convection-diffusion equation.

Write down the one-dimensional heat equation, discretize it with the forward Euler scheme and write down the matrix of the scheme.

Define the global and local truncation error for the instationary convection-diffusion equation.

Formulate Lax's equivalence theorem.

Define zero-stability and absolute stability.

Write down the Fourier series for a grid function  $\varDelta_j$  in d dimensions.

Write down the  $\omega\text{-scheme}.$ 

Which are the interesting values of  $\omega$  for the  $\omega$ -scheme? Why?

Show that the  $\omega$ -scheme is unconditionally stable for  $\omega \geq 1/2.$ 

# 5. The incompressible Navier-Stokes equations

### 5.1 Introduction

In this chapter, the incompressible Navier-Stokes equations in Cartesian coordinates discretized on Cartesian *nonuniform* grids will be considered, discussing most of the basic numerical principles in a simple setting. In practical applications, of course, nonuniform grids and general coordinate systems are prevalent; Discretisation on nonuniform and unstructured grids will be discussed in later chapters.

We can be relatively brief in discussing discretization of the Navier-Stokes equations, because we prepared the ground in our extensive discussion of the convection-diffusion equation in Chapters 2—4. Therefore it wil not be necessary to discuss again the various possibilities for discretizing convection, or stability conditions.

Only the *primitive variable formulation* will be discussed. This means that the velocity components and the pressure will be used as unknowns.

## Purpose of this chapter

The purpose of this chapter is to present a numerical method for the incompressible Navier-Stokes equations. In particular, we will:

- Present suitable boundary conditions;
- Describe spatial discretization on a staggered grid;
- Describe spatial discretization on a colocated grid using the Pressure Weighted Interpolation Method Describe the  $\omega$ -scheme, the Adams-Bashforth scheme and the Adams-Bashforth- $\omega$  scheme for discretization in time:
- Show how to linearize with the Picard or Newton method;
- Describe the pressure-correction method to solve the discretized nonstationary Navier-Stokes equations;
- $\bullet$  Describe the SIMPLE method to solve the discretized stationary Navier-Stokes equations;
- Discuss stability conditions;
- Present some numerical experiments with the MATLAB codes ns1 and ns2;
- $\bullet$  Discuss outflow boundary conditions;
- Discuss efficiency.

# 5.2 Equations of motion and boundary conditions

#### Equations of motion

We restrict ourselves to the two-dimensional case. The equations of motion have been discussed in Chap. 1. For ease of reference, the equations to be considered are repeated here. We assume incompressible flow, i.e.  $D\rho/Dt = 0$ , so that (cf. (1.10))

$$u_{\alpha,\alpha} = 0 (5.1)$$

where we denote partial differentiation by a subscript. The density and viscosity are taken constant. The dimensionless incompressible Navier-Stokes equations are given by equation (1.25):

$$u_{1,t} + u_1 u_{1,1} + u_2 u_{1,2} = -p_{,1} + \text{Re}^{-1} u_{1,\alpha\alpha} ,$$
  
 $u_{2,t} + u_1 u_{2,1} + u_2 u_{2,2} = -p_{,2} + \text{Re}^{-1} u_{2,\alpha\alpha} .$  (5.2)

By adding  $u_1u_{\alpha,\alpha}$  and  $u_2u_{\alpha,\alpha}$  (both zero according to (5.1)), respectively, this can be put in *conservation form*:

$$u_{1,t} + (u_1 u_1)_{,1} + (u_2 u_1)_{,2} = -p_{,1} + \operatorname{Re}^{-1} u_{1,\alpha\alpha} ,$$
  

$$u_{2,t} + (u_1 u_2)_{,1} + (u_2 u_2)_{,2} = -p_{,2} + \operatorname{Re}^{-1} u_{2,\alpha\alpha} .$$
(5.3)

The following units are chosen: velocity: U; length: L; density:  $\rho_0$ ; pressure:  $\rho_0 U^2$ . Then the Reynolds number is given by

$$Re = \rho_0 U L / \mu$$
,

with  $\mu$  the dynamic viscosity coefficient,.

The deviatoric stress tensor (i.e. the viscous part of the stress tensor) is denoted by  $\sigma_{\alpha\beta}$ . From equation (1.17) it follows that

$$\sigma_{11} = 2 \text{Re}^{-1} u_{1,1}$$
  $\sigma_{12} = \sigma_{21} = \text{Re}^{-1} (u_{1,2} + u_{2,1}), \quad \sigma_{22} = 2 \text{Re}^{-1} u_{2,2}.$  (5.4)

The governing equations (5.1) and (5.2) need to be accompanied by initial and boundary conditions.

### Initial conditions

For the momentum equations (5.2) the following initial conditions are required:

$$u(0, \boldsymbol{x}) = u_0(\boldsymbol{x}), \quad v(0, \boldsymbol{x}) = v_0(\boldsymbol{x}),$$

with the prescribed initial velocity field  $u_0$  satisfying the continuity equation (5.1). Note that there is no initial condition for the pressure, since  $p_t$  does not occur.

#### No-slip condition

Viscous fluids cling to solid surfaces. This is called the *no-slip condition*. At a solid surface we have

$$\boldsymbol{u}(t,\boldsymbol{x}) = \boldsymbol{v}(t,\boldsymbol{x}) , \qquad (5.5)$$

with v(t, x) the local wall velocity. The Dirichlet condition (5.5) holds also at open parts of the boundary where the velocity is prescribed, which may be the case at an inflow boundary. But at an inflow boundary one may also prescribe condition (5.6) given below.

#### Free surface conditions

At a free surface the tangential stress components are zero. We consider only the very special case where the free surface is fixed at  $x_2 = a = \text{constant}$ . For the general case, see Sect. 6.2 of Wesseling (2001). At a fixed free surface, the normal velocity and the tangential stress are zero:

$$u_2(t, x_1, a) = 0, \quad u_{1,2}(t, x_1, a) = 0,$$
 (5.6)

where we have used

$$\sigma_{12}(t, x_1, a) = \text{Re}^{-1}(u_{1,2} + u_{2,1})(t, x_1, a) = \text{Re}^{-1}u_{1,2}(t, x_1, a)$$
.

We see that we have a Dirichlet condition for the normal velocity and a Neumann condition for the tangential velocity. A truly free surface moves, its shape must be determined and follows from the condition that the normal stress equals the ambient pressure. This case will not be considered. Conditions (5.6) may also arise at a plane of symmetry. In special cases one may wish to prescribe non-zero tangential stress in (5.6), for example, when one wishes to take the influence of wind shear on a water surface into account.

### Inflow conditions

The momentum equations resemble convection-diffusion equations for  $u_1$  and  $u_2$ , so that the insights gained in the convection-diffusion equation in Chapt. 2—4 provide guidelines for numerical approximation. Based on what we learned about the convection-diffusion equation, we prescribe Dirichlet conditions at an inflow boundary. If, for example,  $x_1 = 0$  is an inflow boundary, we prescribe

$$u_1(t,0,x_2) = U_1(t,x_2) , \quad u_2(t,0,x_2) = U_2(t,x_2) .$$
 (5.7)

### **Outflow conditions**

At an outflow boundary, often not enough physical information is available on which to base a sufficient number of boundary conditions. Usually only the pressure is known. This is not as serious as it may seem, because when  $\text{Re} \gg 1$  'wrong' information generated by an artificial boundary condition propagates

upstream only over a distance of  $\mathcal{O}(\mathrm{Re}^{-1})$ . This is plausible because of the resemblance of (5.2) to the convection-diffusion equation, and may in fact be shown directly by applying singular perturbation analysis to (5.2) in a similar manner as in Sect. 3.2. In order to avoid spurious numerical wiggles it is advisable to choose as artificial outflow condition a homogeneous Neumann condition for the tangential velocity. For an outflow boundary at  $x_1 = a$  this gives:

$$p(t, a, x_2) = p_{\infty}, \quad u_{2,1}(t, a, x_2) = 0.$$
 (5.8)

### Compatibility condition

At every part of the boundary exactly one of the boundary conditions (5.5), (5.6) or (5.8) needs to be prescribed. If it is the case that along the whole of the boundary  $\partial\Omega$  the normal velocity  $u_{\alpha}(t, \boldsymbol{x})n_{\alpha}$  is prescribed, then it follows from (5.1) and the divergence theorem that the following *compatibility condition* must be satisfied:

$$\int_{\partial\Omega} u_a lpha(t, \mathbf{x}) n_a lphadS = 0.$$
 (5.9)

It can be shown theoretically (for further information, see Sect. 6.2 of Wesseling (2001)) that in order for (5.1), (5.2) to be well-posed, the normal component of the prescribed initial velocity field  $u_0(x)$  and a prescribed normal velocity component must match at t = 0:

$$u_{\alpha}^{0}(\boldsymbol{x})n_{\alpha}=u_{\alpha}(0,\boldsymbol{x})n_{\alpha}$$

on parts of  $\partial\Omega$  where the normal velocity is prescribed. But the tangential components of the initial and boundary velocity fields need not match at t=0. Therefore, for example, a sliding wall may be set in motion instantaneously at t=0 in a fluid originally at rest, but one should not let the speed of an arbitrarily shaped body or of an inlet flow change discontinuously.

# 5.3 Spatial discretization on staggered grid

Let the domain be rectangular and be covered with a nonuniform grid consisting of rectangular cells as sketched in Fig. 5.1. The oldest and most straightforward

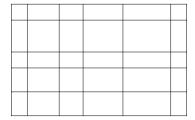
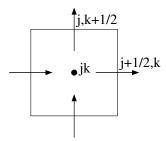


Figure 5.1. Rectangular nonuniform grid

approach to discretizing the Navier-Stokes equations in space is the method proposed in 1965 by Harlow and Welch (see Sect 6.4 of Wesseling (2001) for references to the literature). On orthogonal grids it remains the method of choice.

#### Staggered grid

Grid points for different unknowns are staggered with respect to each other. The pressure resides in the cell centers, whereas the cell face centers contain the normal velocity components, cf. Fig. 5.2. The grid nodes are numbered as



**Figure 5.2.** Staggered placement of unknowns;  $\rightarrow$ ,  $\uparrow$ : velocity components;  $\bullet$ : pressure.

follows. The cell with center at  $x_{ik}$  is called

$$\Omega_{jk}, \quad j=1,\cdots,J, \quad k=1,\cdots,K$$
.

The horizontal and vertical sides of  $\Omega_{jk}$  have length  $h_j^1$  and  $h_k^2$ , respectively. The center of the 'east' side of  $\Omega_{jk}$  is called  $x_{j+1/2k}$ , etc., see Fig. 5.2. Hence,  $\Omega_{jk}$  contains the following unknowns:  $p_{jk}$ ,  $u_{1\ j\pm1/2k}$ ,  $u_{2\ jk\pm1/2}$ . Note that with a staggered grid we always have a mixture of vertex-centered and cell-centered discretization. Unavoidably, at a boundary, some unknowns will have nodes upon it, whereas other unknowns have no nodes on this boundary, but half a mesh size removed. Therefore it is fortunate, as seen in Chapt. 2, that vertex-centered and cell-centered discretization are on equal footing as far as global accuracy and ease of implementation of boundary conditions are concerned.

#### Discretization of continuity equation

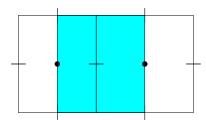
The continuity equation (5.1) is integrated over  $\Omega_{jk}$ , resulting in

$$h_k^2 u_1|_{j-1/2,k}^{j+1/2,k} + h_j^1 u_2|_{j,k-1/2}^{j,k+1/2} = 0$$
 (5.10)

The advantage of the staggered placement of the unknowns is that no further approximation is necessary in this equation.

### Discretization of momentum equations

Finite volume integration takes place over control volumes surrounding  $u_1$  and  $u_2$  grid points, with sides through neighboring pressure points. For example, the control volume for  $u_1_{j+1/2k}$  consists of the union of half of  $\Omega_{j,k}$  and half of  $\Omega_{j+1k}$ , as illustrated in Fig. 5.3. This control volume is called  $\Omega_{j+1/2,k}$ . Finite volume integration gives



**Figure 5.3.** Control volume  $\Omega_{j+1/2,k}$  for  $u_{1,j+1/2k}$ .

$$\int_{\Omega_{j+1/2,k}} \left[ u_{1,t} + (u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1})_{,1} + (u_1 u_2 - \operatorname{Re}^{-1} u_{1,2})_{,2} \right] d\Omega \cong 
h_{j+1/2}^1 h_k^2 \frac{du_{1,j+1/2,k}}{dt} + h_k^2 \left( u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1} \right)_{jk}^{j+1,k} 
+ h_{j+1/2}^1 \left( u_1 u_2 - \operatorname{Re}^{-1} u_{1,2} \right)_{j+1/2,k-1/2}^{j+1/2,k+1/2} = 0 .$$
(5.11)

Here p occurs only in its own nodal points, and further approximation is not necessary. But the derivatives and  $u_1$  and  $u_2$  need to be approximated in terms of surrounding nodes.

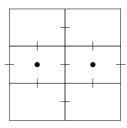
The derivatives are approximated as follows:

$$u_{1,1}|_{jk} \cong (u_{1\ j+1/2,k} - u_{1\ j-1/2k})/h_j^1,$$
  
$$u_{1,2}|_{j+1/2k+1/2} \cong (u_{1\ j+1/2k+1} - u_{1\ j+1/2k})/h_{k+1/2}^2.$$

The central scheme for the inertia term is obtained with the following approximations:

$$u_{1\ jk}^2 \cong (u_{1\ j-1/2,k}^2 + u_{1\ j+1/2,k}^2)/2 ,$$
 
$$(u_1u_2)_{j+1/2,k+1/2} \cong (u_{1\ j+1/2,k} + u_{1\ j+1/2,k+1})(u_{2\ j,k+1/2} + u_{2\ j+1,k+1/2})/4 .$$

The resulting stencil for  $u_{1,j+1/2,k}$  is given in Fig. 5.4.



**Figure 5.4.** Stencil for  $u_{1 j+1/2,k}$ .

The upwind scheme for the inertia term is obtained as follows. We do not wish to test on the sign of  $u_{\alpha}$ , because if statements in for loops are very computer time consuming, cf. Sect. 4.3. We note that upwind approximation of a term  $u\alpha\varphi_{,\alpha}$  can be implemented as follows, without an if statement:

$$u_1 \varphi(x_{j+1/2}) \cong \frac{1}{2} [(u_1 + |u_1|)\varphi_j + (u_1 - |u_1|)\varphi_{j-1}].$$

By using this idea we obtain the following upwind approximation of the inertia terms:

$$u_{1 jk}^{2} \cong \frac{1}{4} \left[ (u_{1} + |u_{1}|)_{j-1/2,k}^{2} + (u_{1} - |u_{1}|)_{j+1/2,k}^{2} \right],$$

$$(u_{1}u_{2})_{j+1/2,k+1/2} \cong \frac{1}{2} \left[ (u_{2} + |u_{2}|)_{j+1/2,k+1/2} u_{1 j+1/2,k} + (u_{2} - |u_{2}|)_{j+1/2,k+1/2} u_{1 j+1/2,k+1} \right],$$

$$(5.12)$$

where

$$u_{2,j+1/2,k+1/2} \equiv (u_{2,j,k+1/2} + u_{2,j+1,k+1/2})/2$$
.

The momentum equation for  $u_2$  is discretized similarly. This completes finite volume discretization in the interior. We continue with the boundary conditions. On the staggered grid the implementation of the boundary conditions (5.5)—(5.8) is just as simple and done in the same way as for the convection-diffusion equation in Chapters 2 and 3.

## The no-slip condition

Let  $x_2 = 0$  be a wall moving horizontally with velocity  $U_1(t)$ . Then the lower side of  $\Omega_{j,1}$  is at the boundary. We have  $u_{2\ j1/2} = 0$ , so that no discretization for  $u_{2\ j,1/2}$  is required. In the finite volume scheme for  $u_{1\ j+1/2,1}$  we need, according to the stencil presented in Fig. 5.4,  $u_{1\ j+1/2,0}$ , which is not available, because  $\boldsymbol{x}_{1\ j+1/2,0}$  is outside the domain. Values outside the domain are called *virtual values*. We write  $u_{1\ j+1/2,0} + u_{1\ j+1/2,1} = 2U(t)$ , so that

$$u_{1 j+1/2,0} = 2U_1(t) - u_{1 j+1/2,1},$$
 (5.13)

which is used to eliminate the virtual value  $u_{1,i+1/2,0}$ .

#### Free surface conditions

Let  $x_2 = a$  be a free surface boundary or a symmetry boundary, so that we have conditions (5.6). Let  $\Omega_{jk}$  be at the boundary. We have  $u_{2j,K+1/2} = 0$ , so that no discrete equation is required for  $u_{2j,K+1/2}$ . In the stencil for  $u_{1j+1/2,K}$  we have  $u_{1j+1/2,K+1}$ , according to Fig. 5.4, which is outside the domain and has to be eliminated. We put  $0 = u_{1,2} \cong (u_{1j+1/2,K+1} - u_{1j+1/2,K})/h_K^2$ , so that

$$u_{1 j+1/2,K+1} = u_{1 j+1/2,K}$$
 (5.14)

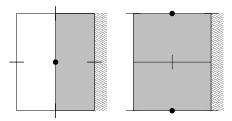
#### Inflow conditions

Let  $x_1 = 0$  be an inflow boundary, so that  $\Omega_{1k}$  is at the boundary. According to (5.7) we have Dirichlet conditions for  $u_1$  and  $u_2$ , so that the situation is almost the same as for the no-slip condition. We put

$$u_{1 \ 1/2,k} = U_1(t, x_{2 \ k}), \quad u_{2 \ 0,k+1/2} = 2U_2(t, x_{2 \ k+1/2}) - u_{2 \ 1,k+1/2}.$$
 (5.15)

#### **Outflow conditions**

Let  $\Omega_{Jk}$  be at an outflow boundary  $x_1 = a$ . We need discrete equations for  $u_{1\ J+1/2,k}$  and  $u_{2\ J,k+1/2}$ . The control volume for  $u_{J+1/2,k}$  consists of half of  $\Omega_{Jk}$ , as illustrated in Fig. 5.5. Finite volume integration gives, similar to (5.11),



**Figure 5.5.** Control volumes for  $u_1$   $_{J+1/2,k}$  and  $u_2$   $_{J,k+1/2}$  at outflow boundary.

$$\int_{\Omega_{J+1/2,k}} \left[ u_{1,t} + (u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1})_{,1} + (u_1 u_2 - \operatorname{Re}^{-1} u_{1,2})_{,2} \right] d\Omega \cong$$

$$\frac{1}{2} h_J^1 h_k^2 (u_1 |_{J+1/2,k})_{,t} + h_k^2 \left( u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1} \right)_{Jk}^{J+1/2,k}$$

$$+ \frac{1}{2} h_J^1 \left( u_1 u_2 - \operatorname{Re}^{-1} u_{1,2} \right)_{J+1/2,k-1/2}^{J+1/2,k+1/2} = 0 .$$

Compared to the interior case, we have to change only terms at or near the outflow boundary. We put the normal stress at  $x_{1 J+1/2,k}$  equal to the prescribed pressure (cf. (5.8)):

$$(p - \text{Re}^{-1}u_{1,1})_{J+1/2,k} = p_{\infty}$$
.

Furthermore,

$$(u_1u_2)_{J+1/2,k+1/2} = u_2 J_{J,k+1/2}u_1 J_{J+1/2,k+1/2}$$

where  $u_{1\ J+1/2,k+1/2}$  is approximated with the upwind scheme or the central scheme. Finally,

$$\mathrm{Re}^{-1}(u_{1,2})_{J+1/2,k+1/2} \cong (u_{1\ J+1/2,k+1} - u_{1\ J+1/2,k})/h_{J+1/2,k+1/2}^2 \ .$$

The scheme for  $u_{2\ J,k+1/2}$  brings nothing new. A virtual value  $u_{2\ J+1,k+1/2}$  occurs as may be seen form the stencil of the  $u_2$ -momentum equation, which is obtained from Fig. 5.4 by rotation over  $90^o$ . This virtual value is eliminated by using  $u_{2,1} = 0$  (cf. (5.8)), which results in

$$u_2 _{J+1,k+1/2} = u_2 _{J,k+1/2}$$

(cf. (5.14)).

### Summary of equations

We put all unknown velocity components in some order in an algebraic vector u and all pressure unknowns in an algebraic vector p, and we divide by

the coefficients of the time derivatives. Then the scheme can be written as a differential-algebraic system of the following structure:

$$\boldsymbol{u}_{.t} + N(\boldsymbol{u}) + G\boldsymbol{p} = \boldsymbol{f}(t) , \qquad D\boldsymbol{u} = \boldsymbol{g}(t) .$$
 (5.16)

Here N is a nonlinear algebraic operator arising from the discretization of the inertia and viscous terms, G is a linear algebraic operator representing the discretization of the pressure gradient, D is a linear algebraic operator representing the discretization of divergence operator in the continuity equation, and f and g are known source terms, arising from the boundary conditions. This is called a differential-algebraic system because it is a mixture of a system of ordinary differential equations (the first member of (5.16)), and algebraic equations (the second member of (5.16)). This completes our description of spatial discretization on the staggered grid.

# 5.4 Spatial discretization on colocated grid

By staggering the unknowns to different positions in the computational cells, a straightforward and accurate discretization on Cartesian grids can be achieved. However, the use of a colocated grid is still popular for a number of different reasons:

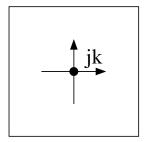
- The extension of a colocated scheme to a structured curvilinear grid can be done easily in a formulation in a Cartesian frame of reference. This is not (that) easily done for a staggered arrangement of unknowns. The latter has certainly been done.
- The compressible Navier-Stokes equations are nearly always discretized on a colocated grid. An algorithm to solve the compressible Navier-Stokes equations can be modified to solve the incompressible equations by including preconditioning on the continuous equations. So the use of a modified algorithm for the compressible Navier-Stokes equations almost automatically leads to a colocated discretization.
- The most efficient solution algorithms for the linear systems resulting from discretization of the momentum and pressure correction equation are based on or make use of hierarchical algorithms like e.g. multigrid. The staggered grid arrangement requires different inter-level operators for each unknown, adding to the complexity of the algorithm.

## Colocated grid

Grid points for different unknowns are colocated with respect to each other. We will discuss the case, where the pressure and both velocity components reside in the cell centers, cf. Fig. 5.6. The grid nodes are numbered as follows. The cell with center at  $x_{jk}$  is called

$$\Omega_{jk}, \quad j=1,\cdots,J, \quad k=1,\cdots,K$$
.

As in the staggered case the horizontal and vertical sides of  $\Omega_{jk}$  have length  $h_j^1$  and  $h_k^2$ , respectively. The center of the 'east' side of  $\Omega_{jk}$  is called  $\boldsymbol{x}_{j+1/2,k}$ , etc., see Fig. 5.6. Hence,  $\Omega_{jk}$  contains the following unknowns:  $p_{jk}$ ,  $u_{1\ jk}$ ,  $u_{2\ jk}$ .



**Figure 5.6.** Colocated placement of unknowns;  $\rightarrow$ ,  $\uparrow$ : velocity components;  $\bullet$ : pressure.

#### Discretization of continuity equation

The continuity equation (5.1) is integrated over  $\Omega_{jk}$ , resulting in

$$h_k^2 u_1|_{j-1/2,k}^{j+1/2,k} + h_j^1 u_2|_{j,k-1/2}^{j,k+1/2} = 0. {(5.17)}$$

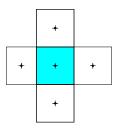
where we use central approximations of  $u_1|^{j+1/2,k}$  and  $u_2|^{j,k+1/2}$  in the following way:

$$u_{1 j+1/2,k} = (u_{1 jk} + u_{1 j+1,k})/2,$$

$$u_{2 j,k+1/2} = (u_{2 jk} + u_{2 j,k+1})/2.$$
(5.18)

# Discretization of momentum equations

Finite volume integration for the momentum equation for the  $u_{\alpha}$  components takes place over the same control volume as used for the discretization of the continuity equation, as illustrated in Fig. 5.7.



**Figure 5.7.** Control volume  $\Omega_{jk}$  for  $u_{1\ jk}$ .

Finite volume integration gives

$$\int_{\Omega_{jk}} \left[ u_{\alpha,t} + (u_{\beta}u_{\alpha} + p - \operatorname{Re}^{-1}u_{\alpha,\beta})_{,\beta} \right] d\Omega \cong 
h_{j}^{1} h_{k}^{2} u_{\alpha jk,t} + h_{k}^{2} (u_{1}u_{\alpha} + \delta_{1\alpha}p - \operatorname{Re}^{-1}u_{\alpha,1})_{j-\frac{1}{2}k}^{j+\frac{1}{2}k} 
+ h_{j}^{1} (u_{2}u_{\alpha} + \delta_{2\alpha}p - \operatorname{Re}^{-1}u_{\alpha,2})_{jk-\frac{1}{2}}^{jk+\frac{1}{2}} = 0 .$$
(5.19)

As opposed to the staggered discretization we need to interpolate the velocities and the pressure for the fluxes.

The derivatives are approximated as follows:

$$u_{\alpha,1}|_{j+1/2,k} \cong (u_{\alpha j+1k} - u_{\alpha jk})/h_j^1,$$
  
 $u_{\alpha,2}|_{j,k+1/2} \cong (u_{\alpha jk+1} - u_{\alpha jk})/h_{k+1/2}^2.$ 

The central scheme for the inertia term is obtained with the following approximations:

$$\begin{array}{rcl} u_{1\ j+1/2,k}^2 &\cong & (u_{1\ jk}^2 + u_{1\ j+1k}^2)/2 \; , \\ (u_1u_2)_{jk+1/2} &\cong & (u_{1\ jk}u_{2\ jk} + u_{1\ j+1k+1}u_{2\ j+1,k+1})/2 \; . \end{array}$$

The resulting stencil for  $u_{1 jk}$  is given in Fig. 5.8.

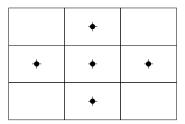


Figure 5.8. Stencil for  $u_{jk}$ .

The first order upwind scheme is obtained as follows:

$$u_{1\ j+1/2k}^2 = u_{1\ jk}^2, \quad u_{1\ jk} + u_{1\ j+1k} \ge 0$$
 (5.20)

$$u_{1\ j+1/2k}^2 = u_{1\ j+1k}^2, \quad u_{1\ jk} + u_{1\ j+1k} < 0$$
 (5.21)

$$(u_1u_2)_{jk+1/2} = (u_1u_2)_{jk}, \quad u_1_{jk} + u_1_{j+1k} \ge 0$$
 (5.22)

$$(u_1u_2)_{jk+1/2} = (u_1u_2)_{jk+1}, \quad u_1_{jk} + u_1_{jk+1} < 0$$
 (5.23)

The pressure is interpolated to the cell faces using a central approximation:

$$p_{j+1/2k} \cong \frac{1}{2} (p_{jk} + p_{j+1k}).$$
 (5.24)

The momentum equation for  $u_2$  is discretized similarly. This completes finite volume discretization in the interior. We continue with the boundary conditions. On the colocated grid the implementation of the boundary conditions (5.5)—(5.8) is just as simple and done in the same way as for the convection-diffusion equation in Chapters 2 and 3.

# The no-slip condition

Let  $x_2 = 0$  be a wall moving horizontally with velocity U(t). Then the lower side of  $\Omega_{j,1}$  is at the boundary. We have  $u_{2,j,1/2} = 0$ . In the finite volume scheme for  $u_{1,j,1}$  we need, according to the stencil presented in Fig. 5.8,  $u_{1,j,0}$ , which is not

available, because  $x_{j,-1}$  is outside the domain. We write  $u_{1j0} + u_{1j1} = 2U_1(t)$ , so that

$$u_{1 \ j0} = 2U_1(t) - u_{1 \ j1} \ , \tag{5.25}$$

which is used to eliminate the virtual value  $u_{1\ j0}$ . Similarly, we need  $u_{2\ j0}$  for the stencil of  $u_{2\ j1}$ . The former can be approximated as:

$$u_{2j0} = -u_{2j1} , (5.26)$$

#### Free surface conditions

Let  $x_2 = a$  be a free surface boundary or a symmetry boundary, so that we have conditions (5.6). Let  $\Omega_{jk}$  be at the boundary. We have  $u_{2\ jK+1/2} = 0$ , that will be used to eliminate the need for  $u_{2\ jK+1}$  in the same way as is done for the no-slip condition. In the stencil for  $u_{1\ jK}$  we have  $u_{1\ jK+1}$ , according to Fig. 5.8, which is outside the domain and has to be eliminated. We put  $0 = u_{1,2} \cong (u_{1\ jK+1} - u_{1\ jK})/h_K^2$ , so that

$$u_{1\ jK+1} = u_{1\ jK},\tag{5.27}$$

$$u_{2jK+1} = -u_{2jK} (5.28)$$

#### Inflow conditions

Let  $x_1 = 0$  be an inflow boundary, so that  $\Omega_{1k}$  is at the boundary. According to (5.7) we have Dirichlet conditions for  $u_1$  and  $u_2$ , so that the situation is almost the same as for the no-slip condition. We put

$$u_{1\ 0k} = 2U_1(t, x_{2\ k}) - u_{1\ 1k}, \quad u_{2\ 0k} = 2U_2(t, x_{2\ k}) - u_{2\ 1k}.$$
 (5.29)

### **Outflow conditions**

Let  $\Omega_{Jk}$  be at an outflow boundary  $x_1 = a$ . We need discrete equations for  $u_{1\ J+1,k}$ . Finite volume integration gives, similar to (5.11),

$$\int_{\Omega_{Jk}} \left[ u_{1,t} + (u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1})_{,1} + (u_1 u_2 - \operatorname{Re}^{-1} u_{1,2})_{,2} \right] d\Omega \cong$$

$$h_J^1 h_k^2 u_{1,t} \,_{J,k} + h_k^2 \left( u_1 u_1 + p - \operatorname{Re}^{-1} u_{1,1} \right)_{J-1/2,k}^{J+1/2,k}$$

$$+ \frac{1}{2} h_J^1 \left( u_1 u_2 - \operatorname{Re}^{-1} u_{1,2} \right)_{J,k-1/2}^{J,k+1/2} = 0 \,.$$

Compared to the interior case, we have to change only terms at or near the outflow boundary. We put the normal stress at  $x_{J+1/2,k}$  equal to the prescribed pressure (cf. (5.8)):

$$(p - \text{Re}^{-1}u_{1,1})_{J+1/2,k} = p_{\infty}$$
.

Finally, we have to specify the value of  $u_{1\ J+1,k}$ , by linear (5.30) or constant (5.31) extrapolation from the interior.

$$u_{1\ J+1k} = 2u_{1\ Jk} - u_{1\ J-1k} \tag{5.30}$$

$$u_{1,J+1k} = u_{1,Jk} \tag{5.31}$$

This concludes our description of the discretization of the incompressible Navier-Stokes equations. However, the discretization just presented is *completely use-less*. We will now show why, by verifying that the following ansatz leads to a solution of the discretized equations (Wesseling (2001)):

$$u_{\alpha jk} = (-1)^{j+k} g(t),$$
 (5.32)

$$p = (-1)^{j+k}, (5.33)$$

where g(t) remains a function to be defined. If we substitute (5.32) and (5.33) in the momentum equation (5.19), the contribution of the inertial terms and the pressure gradient clearly vanish. What remains is the contribution of the diffusive fluxes:

$$h_j^1 h_k^2 (-1)^{j+k} g_t(t) + 4 \frac{h_k^2}{h_j^1} \operatorname{Re}^{-1} g(t) + 4 \frac{h_j^1}{h_k^2} \operatorname{Re}^{-1} g(t) = 0$$

$$g_t(t) + 4 \operatorname{Re}^{-1} \left( \frac{1}{(h_j^1)^2} + \frac{1}{(h_k^2)^2} \right) g(t) = 0, \tag{5.34}$$

The ordinary differential equation (5.34) can be solved for g(t)

$$g(t) = e^{-4\operatorname{Re}^{-1}\left(\frac{1}{(h_j^1)^2} + \frac{1}{(h_k^2)^2}\right)t}$$
(5.35)

Apparently, our ansatz presents a solution that allows so-called checkerboard oscillations for both velocity and pressure, where the former are only slowly dampened for large values of Re and the latter are not dampened at all.

In the early eighties a specific fix was proposed by Rhie and Chow. This fix is commonly referred to as *Using Rhie and Chow* or the *Pressure Weighted Interpolation* method, or PWI-method. Although its use is questionable, it is important to be familiar with this approach and the implications it has for the accuracy of the solution. Other approaches to circumvent the checkerboard modes are discussed in (Wesseling (2001)).

The idea of the PWI method is to replace the linear interpolation of the face velocities in the continuity equation with the following expressions:

$$u_{1 j+1/2,k} = \frac{1}{2} \left( u_{1 jk} + u_{1 j+1,k} \right) + \left( \frac{h_k^2}{4a^1} \Delta^1 p \right) |_{jk}^{j+1,k}, \tag{5.36}$$

$$\Delta^{1} p_{jk} = p_{j+1,k} - 2p_{jk} + p_{j-1,k},$$

$$u_{2\ j,k+1/2} = \frac{1}{2} \left( u_{2\ jk} + u_{2\ j,k+1} \right) + \left( \frac{h_k^1}{4a^2} \varDelta^2 p \right) |_{jk}^{j,k+1}, \eqno(5.37)$$

$$\Delta^2 p_{jk} = p_{j,k+1} - 2p_{jk} + p_{j,k-1},$$

where  $a_{\alpha}$  is the sum of all the coefficients of  $u_{\alpha lm}$ ,  $(lm) \neq (jk)$  (all the off diagonal elements in the discretized momentum equation operator). Because for smooth p,  $\Delta^{1/2}p = \mathcal{O}(h^2)$ , the pressure dependent terms in (5.36) and (5.37) do not influence the order of the interpolation. The discretized continuity equation will now read:

$$h_k^2 u_1|_{j-1/2,k}^{j+1/2,k} + h_j^1 u_2|_{j,k-1/2}^{j,k+1/2} +$$

$$(5.38)$$

$$(h^2)^2 \left\{ \left( \frac{1}{2a^1} \Delta^1 p \right)_{j+1,k} - \left( \frac{1}{a^1} \Delta^1 p \right)_{jk} + \left( \frac{1}{2a^1} \Delta^1 p \right)_{j-1,k} \right\} +$$

$$(h^1)^2 \left\{ \left( \frac{1}{2a^2} \Delta^2 p \right)_{j,k+1} - \left( \frac{1}{a^2} \Delta^2 p \right)_{jk} + \left( \frac{1}{2a^2} \Delta^2 p \right)_{j,k-1} \right\} = 0$$

This corresponds approximately to:

$$u_{\alpha,\alpha} = Ch^4(p_{,1111} + p_{,2222}) \tag{5.39}$$

To have a consistent discretization it is important to use the expressions (5.36) and (5.37) for the convection velocities in the momentum equation and additional convection diffusion equations in the model. We will not discuss the mathematical background of the PWI-method, because there isn't any. The method aims to achieve a coupling between adjacent degrees of freedom in the pressure and this is achieved without affecting the order of accuracy of the discretization.

### Summary of equations

We put all unknown velocity components in some order in an algebraic vector u and all pressure unknowns in an algebraic vector p, and we divide by the coefficients of the time derivatives. Then the scheme can be written as a differential-algebraic system of the following structure:

$$\boldsymbol{u}_{,t} + N(\boldsymbol{u}) + G\boldsymbol{p} = \boldsymbol{f}(t) , \qquad D\boldsymbol{u} + B\boldsymbol{p} = \boldsymbol{g}(t) .$$
 (5.40)

Here N is a nonlinear algebraic operator arising from the discretization of the inertia and viscous terms, G is a linear algebraic operator representing the discretization of the pressure gradient, D is a linear algebraic operator representing the discretization of divergence operator in the continuity equation, B is the discretization of the fourth order operator arising form the Pressure Weighted Interpolation method and f and g are known source terms, arising from the boundary conditions. This is called a differential-algebraic system because it is a mixture of a system of ordinary differential equations (the first member of (5.40)), and algebraic equations (the second member of (5.40)). This completes our description of spatial discretization on the colocated grid.

# 5.5 Temporal discretization on staggered grid

We now discretize also in time. Equation (5.16) is our point of departure. With the forward Euler scheme we obtain:

$$\frac{1}{\tau}(u^n - u^{n-1}) + N(u^{n-1}) + Gp^{n-1/2} = f(t^{n-1}), \qquad Du^n = g(t^n). \quad (5.41)$$

Note that we have to take the pressure term at the new time  $t^n$ , or for better accuracy in time, at  $t^{n-1/2}$ , in order to have degrees of freedom to satisfy the algebraic solenoidality constraint (second equation of (5.41)). In other words,

the pressure always has to be taken implicitly; this is a consequence of the differential-algebraic nature of (5.40). In incompressible flows, the pressure acts as a Lagrange multiplier, that makes it possible to satisfy the continuity equation.

With the  $\omega$ -scheme we obtain:

$$\frac{1}{\tau}(\boldsymbol{u}^{n} - \boldsymbol{u}^{n-1}) + \omega N(\boldsymbol{u}^{n}) + (1 - \omega)N(\boldsymbol{u}^{n-1}) + G\boldsymbol{p}^{n-1/2}$$

$$= \omega \boldsymbol{f}(t^{n}) + (1 - \omega)\boldsymbol{f}(t^{n-1}), \qquad D\boldsymbol{u}^{n} = \boldsymbol{g}(t^{n}).$$
(5.42)

#### Linearization

Equation (5.42) is a nonlinear system, because the nonlinear inertia term makes the operator N nonlinear. To make the system more easily solvable we linearize the operator N. Newton linearization works as follows, writing  $\delta u = u^n - u^{n-1}$ :

$$N(\boldsymbol{u}^n) = N(\boldsymbol{u}^{n-1} + \boldsymbol{\delta u}) \cong N(\boldsymbol{u}^{n-1}) + C(\boldsymbol{u}^{n-1})\boldsymbol{\delta u}$$
,

where C(u) is the Jacobian of N evaluated at u. We clarify this by taking as an example one term of the inertia term (cf. (5.11)):

$$(u_{1\ jk}^{n+1})^2 = (u_{1\ jk}^n + \delta u_{1\ jk})^2 \cong (u_{1\ jk}^n)^2 + 2u_{1\ jk}^n \delta u_{1\ jk}.$$

We eliminate  $\delta u$  and obtain:

$$(u_{1\ ik}^{n+1})^2 \cong 2u_{1\ jk}^n u_{ik}^{n+1} - (u_{1\ jk}^n)^2$$
,

which is linear in the unknown  $u_{1jk}^{n+1}$ . Picard linearization works as follows:

$$(u_{1\ jk}^{n+1})^2 \cong u_{1\ jk}^n u_{1\ jk}^{n+1}$$
.

This is simpler, but temporal accuracy decreases to  $\mathcal{O}(\tau)$ . A second order accurate approximation is obtained if one replaces  $u^n$  by an extrapolation to  $t^{n+1}$ :

$$(u_{1\ jk}^{n+1})^2 \cong (2u_{1\ jk}^n - u_{1\ jk}^{n-1})u_{1\ jk}^{n+1} \ . \tag{5.43}$$

We will call this extrapolated Picard linearization.

After linearization, the matrix C that one obtains changes every time step, because C depends on  $\boldsymbol{u}^{n-1}$ . Matrix generation is computer time consuming. Therefore a time step will be cheaper if the inertia term is discretized explicitly. This gives us a so-called IMEX (implicit-explicit) scheme. The resulting scheme is cheaper, because the implicit operator is now linear and independent of time, so that the corresponding matrix has to be generated only once, and other ingredients necessary for solving, such as an LU factorization, need also to be prepared only once. To maintain second order accuracy in time, it is attractive to use for the explicit part not the forward Euler scheme, but a second order explicit scheme, such as the Adams-Bashforth scheme. For a system of ordinary differential equations  $\boldsymbol{w}_{,t} = \boldsymbol{f}(\boldsymbol{w},t)$  this scheme is given by

$$\frac{1}{\tau}(\boldsymbol{w}^{n} - \boldsymbol{w}^{n-1}) = \frac{3}{2}\boldsymbol{f}(\boldsymbol{w}^{n-1}, t^{n-1}) - \frac{1}{2}\boldsymbol{f}(\boldsymbol{w}^{n-2}, t^{n-2}).$$
 (5.44)

This is called a *two-step method*, because two time steps are involved. At the initial time t=0 one may define  $\mathbf{w}^{-1}=\mathbf{w}^0$ . Application to the Navier-Stokes equations takes place as follows. Let  $N(\mathbf{u})$  in (5.42) be split in a nonlinear inertia part C and a linear viscous part B, as follows:

$$N(\boldsymbol{u}) = C(\boldsymbol{u}) + B\boldsymbol{u} .$$

Then the Adams-Bashforth-Crank-Nicolson scheme is obtained by using (5.42) for  $B\mathbf{u}$  and (5.44) for  $C(\mathbf{u})$ , so that we obtain:

$$\begin{split} \frac{1}{\tau}(\boldsymbol{u}^n - \boldsymbol{u}^{n-1}) + \ &\frac{3}{2}C(\boldsymbol{u}^{n-1}) - \frac{1}{2}C(\boldsymbol{u}^{n-2}) + \frac{1}{2}B(\boldsymbol{u}^n + \boldsymbol{u}^{n-1}) + G\boldsymbol{p}^{n-1/2} \\ &= \boldsymbol{f}(t^{n+1/2}) \ , \qquad D\boldsymbol{u}^n = \boldsymbol{g}(t^n) \ . \end{split}$$

### General formulation on staggered grid

As seen from these examples, time stepping methods applied to equation (5.11) can generally be written as

$$A(\boldsymbol{u}^{n}) + \tau G \boldsymbol{p}^{n-1/2} = \boldsymbol{r}^{n} ,$$
  

$$D\boldsymbol{u}^{n} = \boldsymbol{g}(t^{n}) ,$$
(5.45)

where  $\mathbf{r}^n$  is known from previous time steps and the boundary conditions. For explicit methods, A is the identity I. The system (5.45) is a coupled system for  $\mathbf{u}^n$  and  $\mathbf{p}^{n-1/2}$ . Computing time is reduced if  $\mathbf{p}^{n-1/2}$  and  $\mathbf{u}^n$  can be solved for separately. To this end the following method has been devised, which is the method of choice for nonstationary problems.

#### Pressure-correction method

Equation (5.45) is not solved as it stands, but first a prediction  $u^*$  of  $u^n$  is made that does not satisfy the continuity equation. Then a correction is computed involving the pressure, such that the continuity equation is satisfied. The method is given by:

$$A(\boldsymbol{u}^*) + \tau G \boldsymbol{p}^{n-3/2} = \boldsymbol{r}^n , \qquad (5.46)$$

$$u^n - u^* + \tau G(p^{n-1/2} - p^{n-3/2}) = 0$$
, (5.47)

$$D\boldsymbol{u}^n = \boldsymbol{g}(t^n) \ . \tag{5.48}$$

Equation (5.46) more or less amounts to solving discretized convection-diffusion equations for the predicted velocity components. We use the best available guess for the pressure, namely  $p^{n-3/2}$ . Equation (5.47) is motivated by the fact, that if in the explicit case, where A is the identity, we eliminate  $u^*$  from (5.46) and (5.47), then the original system (5.45) is recovered. The pressure can be computed by applying the operator D to (5.47) and using (5.48), resulting in

$$DG\delta p = \frac{1}{\tau} \{ Du^* - g(t^n) \}, \quad p^{n-1/2} = p^{n-3/2} + \delta p.$$
 (5.49)

After  $p^{n-1/2}$  has been computed,  $u^n$  follows from (5.47).

Equation (5.47) can be regarded as a correction of  $u^*$  for the change in pressure. Therefore (5.46)–(5.49) is called the pressure-correction method. It is an example of a fractional step method, in which a time step is split up in sub-steps, and different physical effects are accounted for separately in the sub-steps. Here pressure forces are accounted for in the second sub-step, and inertia and friction in the first sub-step. Confusingly, the term pressure-correction method is often also applied to various iterative methods to solve the stationary Navier-Stokes equations, in which velocity and pressure updates are carried out not simultaneously but successively. Such methods will be encountered in Chap. 6, where they will be called distributive iteration methods. These should not be confused with the pressure-correction method used in time accurate schemes as formulated above. This method may also be called a projection method, because in (5.47) the new velocity  $u^n$  is the projection of the intermediate velocity field  $u^*$  on the space of velocity fields with discretized divergence equal to zero.

Remembering that divgrad equals the Laplacian, we see that (5.49) looks very much like a discrete Poisson equation; it is frequently called the pressure Poisson equation. Note that no boundary condition needs to be invoked for  $\delta p$  (fortunately, for no such condition is given with the original equations, at least not on the complete boundary), because the boundary conditions have already been taken into account in the construction of D, G and g; the operator DG works exclusively on pressure values at grid points in the interior of the domain.

Even if the method is explicit (A is a diagonal matrix), we still have to solve an implicit system for  $\delta p$ . This is an unavoidable consequence of the differential-algebraic nature of (5.16).

As we remarked before, by elimination of  $u^*$  it is easily seen that in the explicit case the pressure-correction method (5.46)–(5.49) is equivalent to (5.45), and that this remains true if  $p^{n-3/2}$  is neglected in (5.46) and (5.47). But in the implicit case this does not hold, and inclusion of a sufficiently accurate first guess, such as  $p^{n-3/2}$ , for the pressure in (5.46) seems to be necessary to obtain full, i.e.  $\mathcal{O}(\tau^2)$ , temporal accuracy. This may make it necessary to compute the initial pressure field at the starting step (n=1), to be used instead of  $p^{-1/2}$ . This may be done as follows. Application of D to (5.16) at t=0 gives

$$d\mathbf{g}(0)/dt + DN(\mathbf{u}(0)) + DG\mathbf{p}(0) = D\mathbf{f}(0).$$
 (5.50)

After solving p(0) from (5.50), we put  $p^{-1/2} = p(0)$ .

#### Discrete compatibility condition

In case the pressure is not involved in any of the boundary conditions, it follows from the incompressible Navier-Stokes equations that the pressure is determined up to a constant. The system (5.49) for  $\delta p$  is singular in this case, and (5.49) has a solution only if the right-hand side satisfies a compatibility condition. The boundary conditions discussed in Sect. 5.2 are such that if the pressure is not involved in any of the boundary conditions, then the normal velocity component is prescribed all along the boundary, and the compatibility condition (5.9) is

satisfied. Summing the discrete continuity equation (5.10) over all cells reduces, due to cancellation in the interior, to the following sum over boundary points:

$$\sum_{k=1}^{K} h_k^2 u_1 \Big|_{(1/2,k)}^{(J+1/2,k)} + \sum_{j=1}^{J} h_j^1 u_2 \Big|_{(j,1/2)}^{(j,K+1/2)} = 0,$$
 (5.51)

where  $u_1$  and  $u_2$  are the prescribed velocity components at the boundary. If (5.51) is not satisfied exactly one should adjust  $u_1$  and  $u_2$  at the boundaries, which should not be difficult, because of the compatibility condition (5.9). If (5.51) holds, then it turns out that the elements of the right-hand side vector of (5.49) sum to zero, which is precisely the compatibility condition required for existence of solutions of (5.49). If one desires to make the solution unique one can fix  $\delta p$  in some point, but iterative methods usually converge faster if one lets  $\delta p$  float.

#### Temporal accuracy

For literature on the accuracy of the pressure-correction method, see Wesseling (2001), Sect. 6.6 and references quoted there. Indications are that the temporal accuracy of  $\boldsymbol{u}^n$  is of the same order as the order of accuracy of the underlying time stepping method (for example,  $\mathcal{O}(\tau^2)$  for Adams-Bashforth-Crank-Nicolson), but that the accuracy of  $\boldsymbol{p}^{n-1/2}$  is only  $\mathcal{O}(\tau)$ , irrespective of the time stepping method used. If one desires, a pressure field with improved accuracy can be obtained after  $\boldsymbol{u}^n$  has been computed (with the pressure-correction method) by proceeding in the same way as in the derivation of (5.50), leading to the following equation for  $\boldsymbol{p}^{n-1/2}$ :

$$d\mathbf{q}(t^n)/dt + DN(\mathbf{u}^n) + DG\mathbf{p}^{n-1/2} = D\mathbf{f}(t^n),$$

which very likely results in a pressure field  $p^{n-1/2}$  with the same order of temporal accuracy as the velocity field.

#### Stability

For stability of (5.46)—(5.49), it seems necessary that (5.46) is stable. It is conjectured that this is sufficient for the stability of (5.46)—(5.49); numerical evidence supports this conjecture. We restrict ourselves to Fourier stability analysis of (5.46). To this end (5.46) is linearized, the coefficients are taken constant ('frozen'), the boundary conditions are assumed to be periodic, and the known source terms  $\tau G p^{n-3/2}$  and  $r^n$  are neglected. Hence, carrying out Fourier stability analysis for (5.46) implies that the discretization of the following simplified and linearized version of (5.2) is considered:

$$u_{\beta,t} + U_{\alpha}u_{\beta,\alpha} - \text{Re}^{-1}u_{\beta,\alpha\alpha} = 0, \quad \beta = 1, 2.$$
 (5.52)

Equation (5.52) consists of decoupled and identical convection-diffusion equations, for which Fourier stability analysis is presented in Chap. 4. The stability analysis of the Adams-Bashforth-Crank-Nicolson scheme is a bit involved and is not presented in Chap. 4. In Sect. 6.6 of Wesseling (2001) stability conditions

for the Adams-Bashforth-Crank-Nicolson scheme are derived. With the central scheme for the inertia terms we have:

$$\tau \le \max[\tau_1, \min\{\tau_2, \tau_3\}],$$

$$\tau_1 = \frac{4}{3\text{Re}} [U_{\alpha}U_{\alpha}]^{-1},$$

$$\tau_2 = \frac{\text{Re}}{4} [\frac{1}{h_{\alpha}h_{\alpha}}]^{-1},$$

$$\tau_3 = (\frac{3}{\text{Re}})^{1/3} [(U_1^2/h^1)^{2/3} + (U_2^2/h^2)^{2/3}]^{-1}.$$
(5.53)

For the upwind scheme:

$$\tau \leq \max[\tau_{1}, \min\{\tau_{2}, \tau_{3}\}], 
\tau_{1} \equiv \frac{2}{3\text{Re}} \left[ \frac{U_{1}^{2}}{2 + U_{1}h^{1}\text{Re}} + \frac{U_{2}^{2}}{2 + U_{2}h^{2}\text{Re}} \right]^{-1}, 
\tau_{2} \equiv \frac{\text{Re}}{4} \left[ \frac{1 + U_{1}h^{1}\text{Re}/2}{(h^{1})^{2}} + \frac{1 + U_{2}h^{2}\text{Re}/2}{(h^{2})^{2}} \right]^{-1}, 
\tau_{3} \equiv \left( \frac{3}{\text{Re}} \right)^{1/3} \left[ \left( \frac{U_{1}^{4}}{(h^{1})^{2} + U_{1}(h^{1})^{3}\text{Re}/2} \right)^{1/3} + \left( \frac{U_{2}^{4}}{(h^{2})^{2} + U_{2}(h^{2})^{3}\text{Re}/2} \right)^{1/3} \right]^{-1}.$$
(5.54)

The computing cost of checking in every grid point whether the stability condition is satisfied is often not negligible, so in practice this is often done only every 5 or 10 time steps or so, or only once, if one has a reasonable a priori estimate of  $U_1$  (=  $2 \max(u_1)$ ) and  $U_2$  (=  $2 \max(u_2)$ ). Here the factor 2 arises from the nonlinearity of the inertia terms, in the same way as for the Burgers equation in Sect. 4.4.

**Exercise 5.5.1.** What is the Newton linearization of  $u_1u_2$ ?

# 5.6 Numerical experiments

The schemes just described have been implemented in the MATLAB codes ns1 ( $\omega$ -scheme) and ns2 (Adams-Bashforth- $\omega$ -scheme), on a nonuniform Cartesian grid of the type shown in Fig. 5.1. For ns1, Picard linearization or extrapolated Picard linearization is used for the inertia terms. Three types of boundary conditions have been implemented: inflow, outflow and no-slip. The boundaries can be divided in segments, on which one can choose different boundary conditions and numbers of grid cells. For instance, we can do the so-called backward-facing step problem, illustrated in Fig. 5.9. Because the domain is assumed to be rectangular, we cannot handle the narrow inflow part at the left, and use the rectangular domain shown at the right. We choose |AB| = |BC| = 1, |CD| = L, with L to be chosen. It turns out that a recirculation zone is present attached to BC, with length dependent on the Reynolds number Re. The length of the domain L must be larger than the length of the recirculation zone, in order to have  $u_1 > 0$  at DE, so that the outflow boundary condition (5.8) is appropriate. Let the Reynolds number be based on the length of AB and the average inflow velocity.

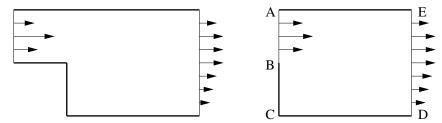


Figure 5.9. The backward-facing step problem.

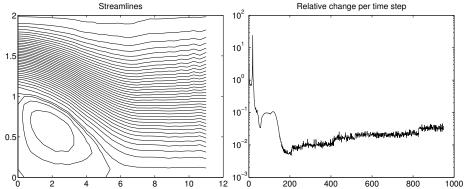
On the segment AB we prescribe inflow:  $u_2 = 0$ ,  $u_1 = f(x_2)$ , f parabolic, such that the average inflow velocity is 1. On DE we prescribe outflow, and on the remaining boundary segments no-slip. The grid is uniform with  $n_1 \times n_2$  cells, where  $n_{\alpha}$  is the grid dimension in the  $x_{\alpha}$  direction.

### Wrong outflow conditions

First, we prescribe Dirichlet conditions, namely a parabolic outflow profile:

$$u_1 = g(x_2), \quad u_2 = 0$$
 (5.55)

on the outflow boundary DE. This is against the advice given concerning outflow boundary conditions in Sect. 5.2. Let us see what goes wrong. Fig. 5.10 shows a result, using Picard linearization. Note that the horizontal and vertical scales are different in the left part of the figure. The relative change per time step is



**Figure 5.10.** Streamlines and convergence history for backward-facing step problem, code ns1, Dirichlet outflow conditions. Re = 100,  $\tau = 0.2, t = 190, n_1 = 30, n_2 = 20, \omega = 1$ , central scheme.

defined as

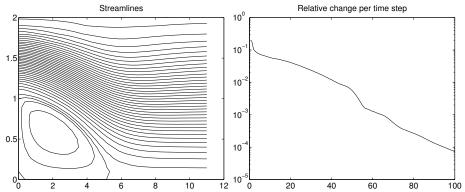
$$\frac{1}{dt} \max \left[ \frac{\parallel \boldsymbol{u}_1^{n+1} - \boldsymbol{u}_1^n \parallel}{q^{n+1}}, \frac{\parallel \boldsymbol{u}_2^{n+1} - \boldsymbol{u}_2^n \parallel}{q^{n+1}}, \frac{\parallel \boldsymbol{p}^{n+1} - \boldsymbol{p}^n \parallel}{\parallel \boldsymbol{p}^n \parallel + \frac{1}{2}(q^{n+1})^2} \right],$$

where  $q = \max(\parallel u_1 \parallel, \parallel u_2 \parallel)$ , and the maximum norm is used. Remember that  $u_{\alpha}$  and p are algebraic vectors containing the (discrete) solution. In the

left part of the figure, we observe unphysical wiggles near the outflow boundary, and the convergence history shows that the solution hesitates to become really stationary. The cause of the wiggles and the remedy have been discussed in Sect. 5.2. For Re=200 a stationary solution was not obtained. Furthermore, it was found that with extrapolated Picard linearization, convergence to a steady solution did not take place.

## Further results on the backward facing step problem

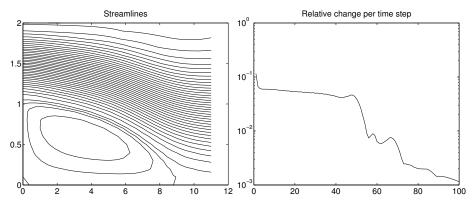
From now on, we use extrapolated Picard linearization. With the correct outflow boundary conditions (5.8) the result shown in Fig. 5.11 is obtained. This



**Figure 5.11.** Streamlines and convergence history for backward-facing step problem, code ns1, outflow conditions(5.8) . Re = 100,  $\tau = 0.6, t = 60, n_1 = 30, n_2 = 20, \omega = 1$ , central scheme.

result looks more satisfactory. The length of the recirculation region agrees with results reported in the literature, and the solution seems to evolve to steady state. Due to the use of Picard linearization, ns1 it is not unconditionally stable; we know no guidelines for choosing the time step  $\tau$ . For linear problems, the  $\omega$ -scheme with  $\omega=1$  is unconditionally stable, as seen in Sect. 4.4.

Fig. 5.12 shows what happens when the Reynolds number is increased. The length of the recirculation region increases; it is thought to be proportional to the Reynolds number. This flow we have also computed with the Adams-Bashforth- $\omega$  scheme (code ns2). The flow pattern obtained is the same as in Fig. 5.12. To determine the time step  $\tau$  we have used the stability criterion for the Adams-Bashforth-Crank-Nicolson scheme (hence,  $\omega=1/2$ ) presented in Sect. 4.4, taking for safety  $\tau$  20% smaller than allowed, giving  $\tau=0.021$ . The convergence history is shown in Fig. 5.13. Although the time required to execute a time step with ns1 is seven times larger than for ns2 (why is it larger, you think?), ns1 is faster because its time step is much larger. The recirculation length in Fig. 5.12 agrees with reports in the literature. With the upwind scheme the recirculation length is found to 7, which is too small. This is because the upwind scheme adds artificial viscosity, and the recirculation length is known



**Figure 5.12.** Streamlines and convergence history for backward-facing step problem, code ns1. Re = 200,  $\tau = 0.6, t = 60, n_1 = 30, n_2 = 20, \omega = 1$ , central scheme.

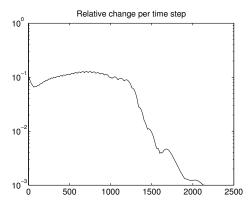


Figure 5.13. Convergence history for backward-facing step problem, code ns2.

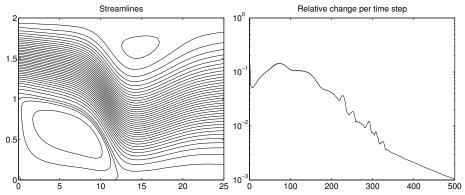
to increase with decreasing viscosity.

The case Re = 400, shown in Fig. 5.14 is found to be more difficult. By trial and error we found that several changes have to be made in order to have the solution converge to steady state:

- 1) The length L has to be increased to make room for the (larger) separation zone and the secondary separation zone that appears at the top wall.
- 2) The vertical number of cells ny has to be increased. But it is found that for stability of ns1 the mesh size ratio  $\Delta x/\Delta y$  should not become too large. Therefore nx also had to be increased.
- 3) The time step has to be decreased.
- 4) The flow takes much longer to settle down to steady state.

As a consequence, Fig. 5.14 takes much more computing time than Fig. 5.12. Again, the recirculation length agrees with the literature.

Several other flow problems have been implemented in the programs  ${\tt ns1}$  and  ${\tt ns2},$  and the reader is invited to experiment with these codes.



**Figure 5.14.** Streamlines and convergence history for backward-facing step problem, code ns1. Re = 400,  $\tau = 0.3$ , t = 150,  $n_1 = 70$ ,  $n_2 = 40$ ,  $\omega = 1$ , central scheme.

## Efficiency

All numerical results in these course notes have been obtained with MATLAB version 5.3 on a Pentium III 550 Mhz processor with 512 MB internal memory. The computation of Fig. 5.14 is the first time that the wall-clock time becomes long enough (10 min) to be annoying. So now we start to get interested in the subject of numerical efficiency. In programming ns1 and ns1 we have followed the MATLAB efficiency guidelines discussed in Sect. 4.2. In order to see where the computational bottlenecks are, we make an inventory of the time spent in various parts of the program.

In ns1 and ns2 all linear systems (of type Ay = b) are solved by MATLAB's direct solver, with the statement

$$y = A b$$

This means that the LU-decomposition of A is computed, and y is found from

$$y = U \setminus (L \setminus b)$$

The pressure-correction matrix does not change during time-stepping, so its LU-decomposition is computed once and stored. The viscous matrix also does not change, because the viscosity is constant. This fact is exploited in code  $\tt ns2$  by means of the IMEX method, in which we have to solve systems with the viscous matrix; so its LU-decomposition is pre-stored as well in  $\tt ns2$ . Of course, we store the matrices as sparse matrices, so that MATLAB exploits sparsity in the execution of  $\tt y = A \ b$ . What consumes most time is the work that has to be done every time step. This time is dominated by the time for solving the linear systems for the velocity components  $t_v$  and the pressure  $t_p$ , by the time for matrix generation (inertia\_matrix in  $\tt ns1$ )  $t_m$ , and by the time for right-hand side generation (right\_hand\_side in  $\tt ns2$ )  $t_r$ . The total time is called  $t_t$ . Table 5.1 gives some runtime statistics for the two cases presented in Figs. 5.12, 5.13 (called case 1 in the table) and 5.14 (case 2).

Let us analyze these figures a bit. The number of unknowns  $(u_1, u_2 \text{ or } p)$  in  $x_1$ direction is  $n_1 + m$  and in  $x_2$ -direction  $n_2 + m$  with m = -1, 0 or 1, depending
on the type of unknown and the type of boundary conditions; of course, for

	ns1	ns1	ns2
	case 1	case 2	case 1
$t_m$	0.039	0.184	_
$t_v$	0.113	0.896	0.009
$t_p$	0.007	0.060	0.007
$t_r$	_	_	0.006
$t_t$	19.2	600	60.1

Table 5.1. Runtime statistics (seconds).

estimating computing work, m can be neglected. This gives us matrices of size  $n \times n$ ,  $n = n_1 n_2$ , with bandwidth, with lexicographic ordering, equal to  $2n_1 - 1$ . It is known that the work to compute the LU-decomposition  $W_{LU}$  and the work to solve a system  $W_S$  with this type of bandmatrix, using standard methods, are given by:

$$W_{LU} \cong 2n_1^3 n_2 \text{flops}, \quad W_S \cong 2n_1^2 n_2 \text{flops}.$$
 (5.56)

(A flop is a floating point operation). We therefore expect

$$t_v \sim n_1^3 n_2$$
,  $t_p \sim n_1^2 n_2$ ,  $t_m \sim n_1 n_2$ .

Between cases 1 and 2 this equation predicts ratios of 25, 11 and 4.7 for  $t_v$ ,  $t_p$  and  $t_m$ , respectively. Obviously, this prediction is much too pessimistic for  $t_v$ , but is not far off the mark for  $t_p$  and  $t_m$ . The reason MATLAB has a  $W_{LU}$  that in our case is much less than given by (5.56) is, that in execution of the command  $y = A \ b$ , if A has been declared to be sparse, the equations are reordered in a clever way, before the triangular factors L and U are computed, in order to reduce computing work. That  $W_{LU}$  may be diminished by reordering is obvious from equation (5.56): by a different numbering of the cells we can interchange the roles of  $n_1$  and  $n_2$  (cf. exercise 5.6.2), which in the present case changes the ratio just predicted for  $t_v$  from 25 to 19. We see that MATLAB does an even better job. Much numerical expertise is hidden behind the  $\$  command in MATLAB.

The table shows that for an increase of  $n \equiv n_1 n_2$  by a factor 4.7, the time required by ns1 for a time step increases by a factor 6.8. The total time  $t_t$  increases by a factor 31. This is mainly due to the much larger number of time steps required. We conclude that for high Reynolds number flows, time stepping is an inefficient way to compute a stationary solution.

Table 5.1 shows that for case 1 the computing time for a time step with ns2 is a factor 7 smaller than with ns1. But because for stability reasons the time step is much smaller, the total computing time  $t_t$  is much larger.

In the next chapter we study other methods to compute stationary solutions, that are hopefully more efficient.

The reader is invited to consult the introduction of this chapter for the topics that we wanted to discuss.

Exercise 5.6.1. In Sect. 3.3 we saw that a Dirichlet outflow condition generates an artificial boundary layer at the outflow boundary. How does the thickness of the boundary layer caused by the wrong outflow condition (5.55) depend on the Reynolds number Re? In Sect. 2.3 we saw that outflow wiggles caused by an outflow Dirichlet condition go away when we apply mesh refinement near the outflow boundary, such the the mesh Péclet number p < 2. Try this for the problem of Fig. 5.10. Define and estimate the mesh Reynolds number, based on the maximum  $u_1$ , which occurs at the inflow boundary. How small should the local mesh size  $\Delta x$  be in the refinement zone? Implement a refinement zone in ns1 by dividing the horizontal domain boundaries in two segments, similar to what is done in ns1 for the vertical boundaries. See what happens.

**Exercise 5.6.2.** In the computation of Fig. 5.14,  $n_1 > n_2$ . Equation (5.56) shows that it would be better if this were the other way around. Specify a vertical backward facing step problem in code **ns1**, and compare run times with the horizontal version.

#### Some self-test questions

Write down the free surface conditions.

What are your preferred outflow conditions, and why?

Discretize the  $x_1$ -momentum equation on a uniform Cartesian grid.

Describe the Adams-Bashforth-Crank-Nicolson scheme.

Write down the general formulation of the discretized nonstationary incompressible Navier-Stokes equations.

Formulate the pressure-correction method.

What is the backward-facing step problem?

Why is the computing time for a time step with the code ns2 smaller than with ns1?

## 6. Discretisation on an unstructured grid.

## 6.1 Introduction

Up till now we have only discussed methods that can solve the Navier-Stokes equations on a Cartesian, structured grid. A structured grid has the property that interior vertices belong to the same number of cells (four in the case of quadrilaterals but other element types would lead to another number). The inherent structure of the structured grid makes the connectivity of cells trivial, and therefore the connectivity does not have to be stored or accessed during the discretisation saving memory and time.

Of course a numerical method that can solve the Navier-Stokes on the unit square or even on a Cartesian geometry is of very limited practical use. We are interested in analyzing the external flow around ships, aircraft, cars or in the internal flow through a pipe with a circular cross section. In all theses cases a Cartesian mesh can not be used because it will not conform to the boundary of the domain of interest. Although it is theoretically possible to use an interpolation scheme to locally adapt the Cartesian mesh to a curvilinear boundary (see for instance the discussion in Vermolen, Verbeek, Vuik, and van Kan (2007), this is seldom done in practice.

Basically, two approaches can be followed, without a specific reference to the discretisation of the equations:

- The use of an unstructured grid consisting of polyhedral control volumes.
- The use of a a body conforming curvilinear grid

In this chapter the first will be briefly discussed, merely to show what is in involved in the generalization from the structured to the unstructured case, while the second approach is introduced in the next chapter. We will limit the discussion to the 2-dimensional case.

# 6.2 Discretisation of the incompressible Navier-Stokes equations on an unstructured grid

We will discuss two examples of the discretisation of the incompressible Navier-Stokes on unstructured grids comprising a set of non-overlapping triangles. Many such approaches exist, but the methods discussed are based on the classic Harlow and Welch staggered grid approach, of which these share many favorable properties, specifically:

- Conservation of mass and momentum
- Conservation of kinetic energy

108

 The absence of spurious checkerboard modes in the pressure without regularization.

The spatial discretisation will be derived for a staggered discretisation based on the *divergence* and the *rotational* form of the Navier-Stokes equations. The latter form is commonly referred to as the *Lamb-Gromeka* form.

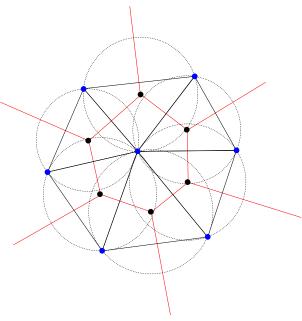
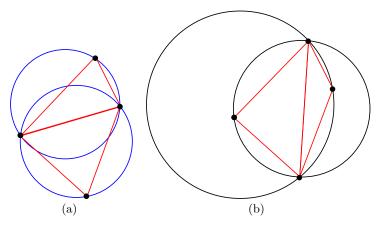


Figure 6.1. A Delaunay triangulation is based on an initial tessellation of *Voronoi tiles* (colored red).

## 6.2.1 Structure in an unstructured grid

For the two discretisation schemes we will discuss, the triangulation is not completely arbitrary: the triangulation is a so-called *Delaunay* triangulation. This triangulation has a number of special properties. The triangulation is constructed from a set of non-overlapping convex polygonal tiles (*Voronoi Tessellation*). The boundary of the Voronoi Tiles consists of line segments, equidistant to all *vertices of the triangles*. The vertices of the polygonal tiles are the circumcenters of the triangles. The circumcircle of each triangle does not contain any other nodes, than the three nodes comprising the triangle. A line connecting the circumcenters of two neighboring triangles intersects the edges of the triangles orthogonally and in the center of the edge. Figure 6.1 shows the Voronoi tiles and the resulting triangulation. The properties of the triangulation significantly simplify the discretisation of the equations. The construction of a Delaunay triangulation is discussed in Chapter\*\*.

## **6.2.2** The rotational form of the incompressible Navier-Stokes equations



**Figure 6.2.** For a *true* Delaunay triangulation, the circumcircle of each triangle does not contain any other nodes than the three nodes of the triangle itself. This is clearly the case for the left (a) but not for the right (b) picture.

$$\mathbf{u}_{,t} + \nabla \cdot (\mathbf{u}\mathbf{u}) = -\nabla p + Re^{-1}\nabla^2 \mathbf{u}$$
(6.1)

The expression **aa** is called a *dyad*, a rank 2 tensor of the following form:

$$\mathbf{ab} = \begin{pmatrix} a_1b_1 & a_1b_2 & a_1b_3 \\ a_2b_1 & a_2b_2 & a_2b_3 \\ a_3b_1 & a_3b_2 & a_3b_3 \end{pmatrix}$$
(6.2)

We can replace the inertia term in the following way:

$$\nabla \cdot (\mathbf{u}\mathbf{u}) = (\nabla \cdot \mathbf{u})\mathbf{u} + (\mathbf{u} \cdot \nabla)\mathbf{u} = (\mathbf{u} \cdot \nabla)\mathbf{u}, \tag{6.3}$$

where the first term in the expansion vanishes because of incompressibility. Using another vector identity the inertia term can be brought in rotational form:

$$\nabla(\mathbf{u} \cdot \mathbf{u}) = 2(\mathbf{u} \cdot \nabla)\mathbf{u} + 2\mathbf{u} \times (\nabla \times \mathbf{u}) \Rightarrow \tag{6.4}$$

$$(\mathbf{u} \cdot \nabla)\mathbf{u} = -\mathbf{u} \times (\nabla \times \mathbf{u}) + \nabla(\frac{1}{2}\mathbf{u} \cdot \mathbf{u})$$
(6.5)

The diffusive term can also be brought in rotational form:

$$\nabla^2 \mathbf{u} = -\nabla \times \nabla \times \mathbf{u} + \nabla(\nabla \cdot \mathbf{u}) = -\nabla \times \nabla \times \mathbf{u}$$
 (6.6)

Introduce the vorticity  $\omega$  and the total pressure  $p^*$  defined as:

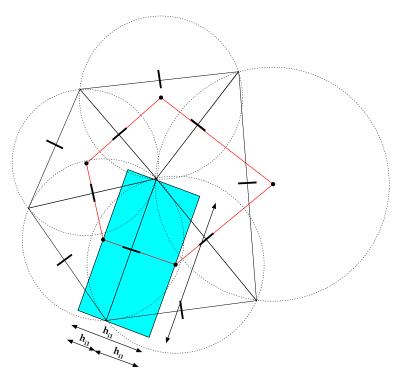
$$\boldsymbol{\omega} = \nabla \times \mathbf{u} \quad p^* = \frac{1}{2} \mathbf{u} \cdot \mathbf{u} + p \tag{6.7}$$

And we end up with the rotational form of the Navier-Stokes equations:

$$\mathbf{u}_{.t} - \mathbf{u} \times \omega = -\nabla p^* - \nabla \times \omega \tag{6.8}$$

In two dimensions the vorticity vector will be:

$$\boldsymbol{\omega} = \hat{\boldsymbol{e}}_3 \omega \tag{6.9}$$



**Figure 6.3.** The control volume used for the discretisation of the Lamb-Gromeka form of the incompressible Navier-Stokes equations.

The equation is projected in the direction of the normal  $\hat{\boldsymbol{n}}_i$  of face i:

$$(u_{n_i})_{,t} - \mathbf{u} \times (\hat{\boldsymbol{e}}_3 \omega) \cdot \hat{\boldsymbol{n}}_i = -(p^*)_{,n_i} - \nabla \times (\hat{\mathbf{e}}_3 \omega) \cdot \hat{\boldsymbol{n}}_i$$
 (6.10)

Define the unit vector tangent  $\hat{s}_i$  to face i as:

$$\hat{\boldsymbol{s}}_{\boldsymbol{i}} = \hat{\boldsymbol{e}}_{\boldsymbol{z}} \times \hat{\boldsymbol{n}} \tag{6.11}$$

Substitution of this expression leads to the following (scalar) equation:

$$(u_{n_i})_{,t} - \omega u_{s_i} = -(p^*)_{,n_i} - \omega_{,s_i}$$
(6.12)

We want to discretize (6.12) either by a finite difference or finite volume formulation, using an arrangement of the variables that corresponds to the Arakawa-C arrangement used by Harlow and Amsden, but now on a unstructured triangular grid. A finite difference formulation will be based on the following information:

- The vorticity  $\omega$  in the edge centers ( the location of the velocities.
- The vorticity  $\omega$  at the nodes of the grid
- The total pressure (= pressure and velocity vector) in the cell circumcenters
- The tangential velocity  $u_s$  at the edge centers.

The vorticity in the nodes is computed using Stokes theorem:

$$\int_{S} \omega dS = \int_{\partial S} \mathbf{u} \cdot d\mathbf{l} \tag{6.13}$$

or discretely:

$$\omega_i = \frac{1}{A_{\text{dual}}} \sum_{T_i \in \text{Conn}_i} u_{s_j} \| \boldsymbol{x}_{c_{T_j}} - \boldsymbol{x}_{c_{T_{j-1}}} \|$$
 (6.14)

The normal derivative of the pressure gradient on edge i can be approximated to second order accuracy by

$$p_{,n}|_{\boldsymbol{x}_{c_{\text{edge}_{i}}}} = \frac{p|_{\boldsymbol{x}_{c_{T_{i}^{+}}}} - p|_{\boldsymbol{x}_{c_{T_{i}^{-}}}}}{\|\boldsymbol{x}_{c_{T_{i}^{+}}} - \boldsymbol{x}_{c_{T_{i}^{+}}}\|}$$
(6.15)

$$\omega|_{\boldsymbol{x}_{c_{\text{edge}_{i}}}} = \frac{1}{2} \left( \omega|_{\boldsymbol{x}_{\text{edge}_{i_{\text{node1}}}}} + \omega|_{\boldsymbol{x}_{\text{edge}_{i_{\text{node2}}}}} \right)$$
 (6.16)

## 6.2.3 The divergence form of the Navier-Stokes equations

Alternatively, the divergence form of the Navier-Stokes equations can be discretised on an unstructured triangular grid. Start from the Navier-Stokes equations vector form (6.1) and multiply with the local normal vector  $\hat{n}_i$ :

$$\hat{\boldsymbol{n}}_{i} \left( \boldsymbol{u}_{.t} + \boldsymbol{\nabla} \cdot (\boldsymbol{u} \boldsymbol{u}) \right) = \left( \nabla p + \operatorname{Re}^{-1} \boldsymbol{\nabla}^{2} \boldsymbol{u} \right) \hat{\boldsymbol{n}}_{i}$$
 (6.17)

The equation (6.17) is integrated over the controlvolume depicted in Fig. 6.4. Note that the boundary of the controlvolume does not coincide with the cell interfaces, and that the control volumes are actually overlapping. However, neither of these properties will adversely affect the accuracy of the discretisation.

$$\int_{E} (u_{i})_{,t} + \hat{\boldsymbol{n}} \cdot (\boldsymbol{\nabla} \cdot (\boldsymbol{u}\boldsymbol{u})) dS = \int_{E} (-\boldsymbol{\nabla} p + \operatorname{Re}^{-1} \boldsymbol{\nabla}^{2} \boldsymbol{u}) \cdot \hat{\boldsymbol{n}}_{i}$$
(6.18)

The inertial term can be expressed in the following way:

$$\int_{E} \nabla \cdot (\boldsymbol{u}\boldsymbol{u}) dS = h_1 l_e \boldsymbol{c_1} + h_2 l_e \boldsymbol{c_2}, \tag{6.19}$$

where  $c_1$  is defined as:

$$c_{i} = \frac{1}{A_{T_{i}}} \int_{T_{i}} \nabla \cdot (\boldsymbol{u}\boldsymbol{u}) dS. \tag{6.20}$$

Apply the divergence theorem to formulate (6.20) as a boundary integral:

$$\frac{1}{A_{T_i}} \int_{T_i} \nabla \cdot (\boldsymbol{u}\boldsymbol{u}) dS = \frac{1}{A_{T_i}} \int_{\partial T_i} (\boldsymbol{u}\boldsymbol{u}) \cdot \hat{\boldsymbol{n}} d\Gamma 
= \frac{1}{A_{T_i}} \int_{\partial T_i} \boldsymbol{u} u_n d\Gamma = \frac{1}{A_{T_i}} \sum_{\text{edges}_{T_i}} (\boldsymbol{u} u_n)|_{\text{edge k}}$$
(6.21)

The velocity vector  $\boldsymbol{u}$  in the center of the edge is approximated as:

$$\boldsymbol{u}_{\mathrm{edge k}} = \frac{1}{2} \left( \boldsymbol{u}_{T_{\mathrm{edge k}}^{+}} + \boldsymbol{u}_{T_{\mathrm{edge k}}^{-}} \right),$$
 (6.22)

where  $u_T$  denotes the velocity vector in the circumcenter of triangle T. Because of the Delaunay triangulation we can approximate the integral over the pressure gradient term in the following way:

$$\int_{E} (-\boldsymbol{\nabla} p) \,\hat{\boldsymbol{n}}_{i} dS = l_{i} p |_{\boldsymbol{x}_{c_{T}-}}^{\boldsymbol{x}_{c_{T}+}}$$

$$(6.23)$$

The diffusion term is expressed very similarly to the inertial term:

$$\int_{E} \operatorname{Re}^{-1} \nabla^{2} \boldsymbol{u} dS = l_{i} h_{1} \boldsymbol{d}_{1} + l_{i} h_{2} \boldsymbol{d}_{2}, \tag{6.24}$$

where

$$d_{i} = \frac{1}{A_{T_{i}}} \int_{T_{i}} \operatorname{Re}^{-1} \nabla^{2} \boldsymbol{u} dS = \frac{1}{A_{T_{i}}} \int_{T_{i}} \operatorname{Re}^{-1} \nabla \cdot (\nabla \boldsymbol{u}) dS =$$

$$\frac{1}{A_{T_{i}}} \int_{\partial T_{i}} \operatorname{Re}^{-1} \frac{\partial \boldsymbol{u}}{\partial n} d\Gamma = \frac{\operatorname{Re}^{-1}}{A_{T_{i}}} \sum_{\text{edges}} \frac{\boldsymbol{u}^{+} - \boldsymbol{u}^{-}}{h_{i}}$$

$$(6.25)$$

Combining the expressions for  $c_i$ ,  $d_i$  and the discrete pressure gradient, the discrete momentum equation reads:

$$hl_i(u_i)_{,t} + \hat{\boldsymbol{n}}_i l_e \left( h_1 \boldsymbol{c}_1 + h_2 \boldsymbol{c}_2 \right) = - l_i p |_{\boldsymbol{x}_{c_{T^-}}}^{\boldsymbol{x}_{c_{T^+}}} + \hat{\boldsymbol{n}}_i l_i \left( h_1 \boldsymbol{d}_1 + h_2 \boldsymbol{d}_2 \right)$$
 (6.26)

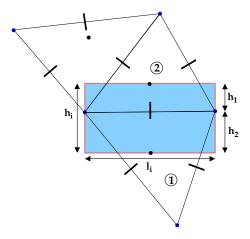


Figure 6.4.

#### Some self-test questions

What is the reason for choosing an unstructured discretisation?

The controlvolumes for the discretisation of the momentum equations are overlapping. Does this violate global momentum conservation?

## 7. Discretisation on a boundary conforming grid.

## 7.1 The basic idea

The basic idea of the approach is the following. The previous chapters describe the discretisation of the Navier-Stokes equations (or the convection-diffusion equation for that matter) on a uniform Cartesian grid: a grid that consists of rectangular cells of uniform width and height. Of course the applicability of a method that can solve the before mentioned equations on a geometry that can be subdivided into such cells is very limited. We will now define a one-to-one mapping  $x(\xi)$  that transforms a rectangular region in the Computational domain to an arbitrarily shaped region in the *Physical* domain. This very mapping is used to transform the equations we intended to solve such that the dependent variables will be in the computational domain: they are no longer expressed in Cartesian coordinates but in so-called general coordinates. You might expect that it would be more helpful to have the inverse mapping  $\boldsymbol{\xi}(\boldsymbol{x})$ , but it turns out it will generally not be easy to construct this inverse mapping. Fortunately, it turns out is sufficient to know that this inverse mapping exists (the original mapping is one-to-one and onto), and its use can be completely avoided. This approach will allow us to discretize the transformed equations on a Cartesian grid, using the techniques discussed in the previous chapters. The mapping itself will generally not be available as  $x(\xi)$ , but as a grid function  $\xi_{ij} \to x_{ij}$ , for which the derivatives (required for the formulation in general coordinates) can be accurately approximated using finite differences as long as the grid is sufficiently smooth. The key idea is therefore that our transformation makes the

## 7.2 Some tensor calculus

domain and discretisation.

To be able to formulate the equations in general coordinates, we need to understand some basic tensor algebra. Only the very basics will be discussed here. If you are interested in continuum mechanics or electromagnetics, a full course in tensor calculus (WI4037TU, second semester, Daalderop) is indispensable. Note that in the current context, the summation convention applies to combinations of matched super and subscripts. Why this is not in contradiction with the Cartesian case, where we applied to summation convention to matching sets of two matching super or subscripts will become clear shortly.

equations 'more complicated' in exchange for a 'less complicated' computational

## 7.3 The coordinate mapping

## 7.3.1 Definition of the basis vectors

We will use a coordinate system based on a *covariant* and *contravariant* basis. The meaning of these names will be explained shortly.

The vectors  $\mathbf{a}_{(\alpha)}$  comprising the covariant basis and the vectors  $\mathbf{a}^{(\alpha)}$  that comprise the contravariant basis are defined as, respectively:

$$a_{(\alpha)} \equiv \frac{\partial x}{\partial \xi^{\alpha}}, \quad a^{(\alpha)} \equiv \nabla \xi^{\alpha}$$
 (7.1)

$$(\boldsymbol{a}_{(\alpha)})^{\beta} = \frac{\partial x^{\beta}}{\partial \xi^{\alpha}}, \quad (\boldsymbol{a}^{(\alpha)})_{\beta} = \frac{\partial \xi^{\alpha}}{\partial x^{\beta}}$$
 (7.2)

In words: the covariant base vectors are the partial derivatives of the coordinate mapping with respect to the respective curvilinear coordinates, while the contravariant basis vectors are the gradient vectors of the inverse coordinate mapping  $\boldsymbol{\xi}(\boldsymbol{x})$  Note that the vectors that comprise the covariant basis are easily approximated from discrete values of the mapping  $\boldsymbol{x}(\boldsymbol{\xi})$  using finite differences, but that direct approximation of the elements of the contravariant basis would require evaluation of the *inverse* mapping  $\boldsymbol{\xi}(\boldsymbol{x})$ . The covariant vectors are denoted with subscripts ('co' rhymes with 'low', but this might not be the reason). Neither the contravariant nor the covariant basis is (necessarily) orthogonal. Furthermore, the vectors that comprise the basis do not have unit length, as is the case in the Cartesian standard basis:

$$|\boldsymbol{a}_{(\alpha)}| \neq 1, \quad |\boldsymbol{a}^{(\alpha)}| \neq 1$$
 (7.3)

However it is easy to show that:

$$\mathbf{a}_{(\alpha)} \cdot \mathbf{a}^{(\beta)} = \begin{cases} 0, & \alpha \neq \beta \\ 1, & \alpha = \beta \end{cases}$$
 (7.4)

Because:

$$\boldsymbol{a}_{(\alpha)} \cdot \boldsymbol{a}^{(\beta)} = \frac{\partial \xi^{\alpha}}{\partial x^{\gamma}} \frac{\partial x^{\gamma}}{\partial \xi^{\beta}} = \frac{\partial \xi^{\alpha}}{\partial \xi^{\beta}} = \delta^{\alpha}_{\beta}$$
 (7.5)

Relation (7.4) will be used later to compute the vectors in the contravariant basis from the vectors in the covariant basis.

## 7.3.2 Construction of the basis vectors

In practice we will construct a mapping  $x(\xi)$ , either numerically or analytically. This means the covariant base vectors are easily computed, using (finite) differencing of the mapping. However, the contravariant base vectors are not that easily found, because their direct computation would require the inverse mapping  $\xi(x)$ , which will generally not be available. Fortunately, we can use relation (7.4). The following should obviously hold:

$$\mathbf{a}^{(1)} \cdot \mathbf{a}_{(\beta)} = \delta_{\beta}^{1}, \quad \beta = 1, 2, 3.$$
 (7.6)

As an example we will show how the compute  $a_{(1)}$  from a known covariant basis. This relation (7.6) can be cast in matrix form to read:

$$\begin{pmatrix} \boldsymbol{a}_{(1)}^T \\ \boldsymbol{a}_{(2)}^T \\ \boldsymbol{a}_{(3)}^T \end{pmatrix} \boldsymbol{a}^{(1)} = \hat{\boldsymbol{e}}_{(1)}, \tag{7.7}$$

where the superscript T denotes the transpose of the vector it is applied to. The system (7.7) can be solved for the contravariant base vector  $\boldsymbol{a}^{(1)}$  using Cramers Rule. This requires evaluation of the determinants of four matrices:

$$\det \begin{pmatrix} \boldsymbol{a}_{(1)}^T \\ \boldsymbol{a}_{(2)}^T \\ \boldsymbol{a}_{(3)}^T \end{pmatrix} = \det(\boldsymbol{a}_{(1)} \ \boldsymbol{a}_{(2)} \ \boldsymbol{a}_{(3)}) = \boldsymbol{a}_{(1)} \cdot (\boldsymbol{a}_{(2)} \times \boldsymbol{a}_{(3)}) = J = \sqrt{g}$$
 (7.8)

$$\det \begin{pmatrix} \boldsymbol{e}_{(1)}^T \\ \boldsymbol{a}_{(2)}^T \\ \boldsymbol{a}_{(3)}^T \end{pmatrix} = \det(\hat{\boldsymbol{e}}_{(1)} \ \boldsymbol{a}_{(2)} \ \boldsymbol{a}_{(3)}) = \hat{\boldsymbol{e}}_{(1)} \cdot (\boldsymbol{a}_{(2)} \times \boldsymbol{a}_{(3)})$$
(7.9)

$$\det \begin{pmatrix} e_{(2)}^T \\ a_{(2)}^T \\ a_{(3)}^T \end{pmatrix} = \det(\hat{e}_{(2)} \ a_{(2)} \ a_{(3)}) = \hat{e}_{(2)} \cdot (a_{(2)} \times a_{(3)})$$
(7.10)

$$\det \begin{pmatrix} \boldsymbol{e}_{(3)}^T \\ \boldsymbol{a}_{(2)}^T \\ \boldsymbol{a}_{(3)}^T \end{pmatrix} = \det(\hat{\boldsymbol{e}}_{(3)} \ \boldsymbol{a}_{(2)} \ \boldsymbol{a}_{(3)}) = \hat{\boldsymbol{e}}_{(3)} \cdot (\boldsymbol{a}_{(2)} \times \boldsymbol{a}_{(3)})$$
(7.11)

These relations can be derived when the expressions for the determinants are expanded, they are not (completely) obvious. The reason why we denote the Jacobian of the mapping J by  $\sqrt{g}$  will become clear shortly.

In resume the first vector of the contravariant basis can be expressed as:

$$a^{(1)} = \frac{1}{\sqrt{q}} \left( a_{(2)} \times a_{(3)} \right)$$
 (7.12)

Of course, a similar relation can be derived for the other contravariant base vectors. It can be shown that the following general expression holds for all three contravariant base vectors:

$$\mathbf{a}^{(\alpha)} = \frac{1}{\sqrt{g}} \left( \mathbf{a}_{(\beta)} \times \mathbf{a}_{(\gamma)} \right), \quad \alpha, \beta, \gamma \text{ cyclic even permutation of } 1,2,3$$
 (7.13)

There is an alternative way to come to this expression. Note that (7.4) makes  $\boldsymbol{a}^{(\alpha)}$  orthogonal to both  $\boldsymbol{a}_{(\beta)}$  and  $\boldsymbol{a}_{(\gamma)}$ . Therefore,  $\boldsymbol{a}^{(\alpha)}$  will be in the direction of  $\boldsymbol{a}_{(\beta)} \times \boldsymbol{a}_{(\gamma)}$ , but with length to be determined:

$$\boldsymbol{a}^{(\alpha)} = C\left(\boldsymbol{a}^{(\beta)} \times \boldsymbol{a}^{(\gamma)}\right),\tag{7.14}$$

where C denotes the appropriate, scaling factor that still has to be determined. Now pre-multiply (7.14) on both sides with  $\mathbf{a}_{(\alpha)}$  and solve for C:

$$C = \frac{1}{\boldsymbol{a}_{(\alpha)} \cdot (\boldsymbol{a}^{(\beta)} \times \boldsymbol{a}^{(\gamma)})} = \frac{1}{J} = \frac{1}{\sqrt{g}}$$
 (7.15)

As an example, the cylindrical coordinate system is used to demonstrate the computation of the vectors in the covariant and contravariant basis. This coordinate mapping  $x(\xi)$ ;  $x = (x_1, x_2, x_3)$ ,  $\xi = (r, \theta, z)$  is given by:

$$x_1 = r \cos \theta$$

$$x_2 = r \sin \theta$$

$$x_3 = z$$

$$(7.16)$$

The vectors in the covariant basis are given by:

$$\mathbf{a}_{(r)} = \left(\frac{\partial x_1}{\partial r}, \frac{\partial x_2}{\partial r}, \frac{\partial x_3}{\partial r}\right) = (\cos \theta, \sin \theta, 0)$$
 (7.17)

$$\mathbf{a}_{(\theta)} = \left(\frac{\partial x_1}{\partial \theta}, \frac{\partial x_2}{\partial \theta}, \frac{\partial x_3}{\partial \theta}\right) = (-r\sin\theta, r\cos\theta, 0)$$
 (7.18)

$$\mathbf{a}_{(z)} = \left(\frac{\partial x_1}{\partial z}, \frac{\partial x_2}{\partial z}, \frac{\partial x_3}{\partial z}\right) = (0, 0, 1)$$
 (7.19)

Next we compute the vectors in the contravariant basis:

$$\mathbf{a}^{(r)} = \frac{1}{\sqrt{g}} \left( \mathbf{a}_{(\theta)} \times \mathbf{a}_{(z)} \right) = \frac{1}{r} \begin{vmatrix} \hat{\mathbf{e}}_{(1)} & \hat{\mathbf{e}}_{(2)} & \hat{\mathbf{e}}_{(3)} \\ -r\sin\theta & r\cos\theta & 0 \\ 0 & 0 & 1 \end{vmatrix} = (\cos\theta, \sin\theta, 0) \quad (7.20)$$

$$\mathbf{a}^{(\theta)} = \frac{1}{\sqrt{g}} \left( \mathbf{a}_{(z)} \times \mathbf{a}_{(r)} \right) = \frac{1}{r} \begin{vmatrix} \hat{\mathbf{e}}_{(1)} & \hat{\mathbf{e}}_{(2)} & \hat{\mathbf{e}}_{(3)} \\ 0 & 0 & 1 \\ \cos \theta & \sin \theta & 0 \end{vmatrix} = \frac{1}{r} \left( -\sin \theta, \cos \theta, 0 \right) \quad (7.21)$$

$$\boldsymbol{a}^{(z)} = \frac{1}{\sqrt{g}} \left( \boldsymbol{a}_{(r)} \times \boldsymbol{a}_{(\theta)} \right) = \frac{1}{r} \begin{vmatrix} \hat{\boldsymbol{e}}_{(1)} & \hat{\boldsymbol{e}}_{(2)} & \hat{\boldsymbol{e}}_{(3)} \\ \cos \theta & \sin \theta & 0 \\ -r \sin \theta & r \cos \theta & 0 \end{vmatrix} = (0, 0, 1)$$
(7.22)

It is straightforward to verify that (7.6) holds for the vectors in the contravariant and covariant basis. Note that these vectors have different length.

**Exercise 7.3.1.** Derive the covariant and contravariant basis for the spherical coordinate system:  $x(\xi)$ ;  $x = (x_1, x_2, x_3)$ ,  $\xi = (r, \theta, \phi)$  is given by:

$$x_1 = r \cos \theta \sin \varphi,$$

$$x_2 = r \sin \theta \sin \varphi,$$

$$x_3 = r \cos \theta.$$
(7.23)

#### 7.3.3 Covariant and contravariant vectors

What do the terms *covariant* and *contravariant* mean? It has to do with the way a vector field transforms when you are changing to a different coordinate system. Assume you have two different curvilinear coordinate systems,  $eta = (\eta_1, \eta_2, \eta_3)$ 

and  $\boldsymbol{\xi} = (\xi_1, \xi_2, \xi_3)$ . Assume a given vector field is expressed in its components, and changing from the  $\boldsymbol{\xi}$  coordinate system to the  $\boldsymbol{eta}$  coordinate system, these components transform in the following way:

$$\tilde{U}^{\beta}(\boldsymbol{\eta}) = \frac{\partial \eta^{\beta}}{\partial \boldsymbol{\xi}^{\alpha}} U^{\alpha}(\boldsymbol{\xi}). \tag{7.24}$$

Then the vector field is a *contravariant* vector field, as indicated by the components being written down as superscripts. On the other hand, if the components transform according to the following transformation rule:

$$\tilde{U}_{\beta}(\boldsymbol{\eta}) = \frac{\partial \xi^{\alpha}}{\partial n^{\beta}} U_{\alpha}(\boldsymbol{\xi}). \tag{7.25}$$

Then the vector field is a *covariant* vector field, as indicated by the components being written down as subscripts. For the Cartesian case, there is no distinction between the covariant and contravariant components. Therefore, we can use either notation. To distinguish between a Cartesian and a curvilinear vector the former is written down as a lower case letter, as opposed to an upper case letter.

## 7.3.4 Representation of u

Starting from the Cartesian basis, we can express the velocity using a contravariant basis. Recall that a contravariant entity transforms in the following way:

$$\tilde{U}^{\beta}(\boldsymbol{\eta}) = \frac{\partial \eta^{\beta}}{\partial \xi^{\alpha}} U^{\alpha}(\boldsymbol{\xi}) \tag{7.26}$$

We apply this transformation rule to transform a vector expressed in its Cartesian components to a vector in curvilinear, contravariant components:

$$U^{\beta} = \frac{\partial \xi^{\beta}}{\partial x^{\alpha}} u^{\alpha} = \boldsymbol{a}_{\alpha}^{(\beta)} u^{\alpha} = \boldsymbol{a}^{(\beta)} \cdot \boldsymbol{u}$$
 (7.27)

These  $U^{\beta}$  are the contravariant components of the velocity vector  $\boldsymbol{u}$ . Conversely, we apply the transformation rule to transform a vector expressed in curvilinear, contravariant components to a vector in its Cartesian components:

$$u^{\beta} = \frac{\partial x^{\beta}}{\partial \xi^{\alpha}} U^{\alpha} = (\boldsymbol{a}_{(\alpha)})^{\beta} U^{\alpha}$$
 (7.28)

The full vector  $\boldsymbol{u}$  is then given by:

$$\boldsymbol{u} = \boldsymbol{a}_{(\alpha)} U^{\alpha}. \tag{7.29}$$

This is a representation of the vector  $\boldsymbol{u}$  in the co-variant basis vectors and contravariant components. On the other hand the same vector  $\boldsymbol{u}$  can be expressed using a set of contravariant basis vectors and co-variant components. Recall that a covariant entity transforms in the following way:

$$\tilde{U}_{\beta}(\boldsymbol{\eta}) = \frac{\partial \boldsymbol{\xi}^{\alpha}}{\partial \eta^{\beta}} U_{\alpha}(\boldsymbol{\xi}) \tag{7.30}$$

We apply this transformation rule to transform a vector expressed in its Cartesian components to a vector in curvilinear, covariant components:

$$U_{\beta} = \frac{\partial x^{\alpha}}{\partial \xi^{\beta}} u_{\alpha} = \boldsymbol{a}_{(\beta)}^{\alpha} u_{\alpha} = \boldsymbol{a}_{(\beta)} \cdot \boldsymbol{u}$$
 (7.31)

These are the covariant components of the velocity vector  $\boldsymbol{u}$ . Conversely, we apply the transformation rule to transform a vector expressed in curvilinear, covariant components to a vector in its Cartesian components:

$$u_{\beta} = \frac{\partial \xi^{\alpha}}{\partial x^{\beta}} U_{\alpha} = (\boldsymbol{a}^{(\alpha)})_{\beta} U_{\alpha} \tag{7.32}$$

The full vector  $\boldsymbol{u}$  is then given by:

$$\boldsymbol{u} = \boldsymbol{a}^{(\alpha)} U_{\alpha}. \tag{7.33}$$

This is a representation of the vector  $\boldsymbol{u}$  in the *contra*-variant basis vectors and *co*-variant components.

## 7.3.5 The permutation symbol $\varepsilon_{\alpha\beta\gamma}$

One way to define the vector product is by use of the permutation symbol  $\varepsilon_{\alpha\beta\gamma}$ . The permutation symbol is defined in the following way:

$$\varepsilon_{\alpha\beta\gamma} \equiv \begin{cases} 0 &, & \text{if any of } \alpha, \beta, \gamma \text{ are equal} \\ 1 &, & \alpha, \beta, \gamma \text{ even permutation of } 1, 2, 3 \\ -1 &, & \alpha, \beta, \gamma \text{ odd permutation of } 1, 2, 3 \end{cases}$$
 (7.34)

Some examples:

$$\varepsilon_{122} = 0 \tag{7.35}$$

$$\varepsilon_{231} = 1, \quad 231 \rightarrow 321 \rightarrow 123 \text{ (double permutation)}$$
 (7.36)

$$\varepsilon_{213} = -1 \quad 213 \to 123 \text{ (single permutation)}$$
 (7.37)

The vector product can now be defined as (check that this holds indeed):

$$c = a \times b, \quad c_{\alpha} = \varepsilon_{\alpha\beta\gamma} a^{\beta} b^{\gamma}$$
 (7.38)

And the vector triple product can be defined as:

$$\mathbf{a}_{(\alpha)} \cdot (\mathbf{a}_{(\beta)} \times \mathbf{a}_{(\gamma)}) = \varepsilon_{\delta\sigma\mu}(\mathbf{a}_{(\alpha)})^{\delta}(\mathbf{a}_{(\beta)})^{\sigma}(\mathbf{a}_{(\gamma)})^{\mu}$$
 (7.39)

Like vectors, tensors are also classified by the way they transform from one to another coordinate system. A tensor  $T_{\alpha\beta\gamma}^{\kappa\mu}(\boldsymbol{\xi})$  is a tensor of covariant order three, contravariant order two and weight w, if it transforms according to the following transformation rule:

$$\tilde{T}^{\sigma\rho}_{\nu\omega\psi}(\boldsymbol{\eta}) = (\sqrt{g})^w \frac{\partial \xi^\alpha}{\partial \eta^\nu} \frac{\partial \xi^\beta}{\partial \eta^\omega} \frac{\partial \xi^\gamma}{\partial \eta^\omega} \frac{\partial \eta^\sigma}{\partial \eta^\omega} \frac{\partial \eta^\rho}{\partial \xi^\mu} T^{\kappa\mu}_{\alpha\beta\gamma}(\boldsymbol{\xi})$$
(7.40)

The permutation symbol is a tensor of covariant order three and weight -1. This is easy to verify:

$$\tilde{\varepsilon}_{\alpha\beta\gamma} = \left| \frac{\partial \mathbf{x}}{\partial \boldsymbol{\xi}} \right|^{w} \frac{\partial x^{\delta}}{\partial \xi^{\alpha}} \frac{\partial x^{\sigma}}{\partial \xi^{\beta}} \frac{\partial x^{\mu}}{\partial \xi^{\gamma}} \varepsilon_{\delta\sigma\mu} =$$

$$(\sqrt{g})^{w} \mathbf{a}_{(\alpha)} \cdot (\mathbf{a}_{(\beta)} \times \mathbf{a}_{(\gamma)}) = (\sqrt{g})^{w+1} \varepsilon_{\alpha\beta\gamma}$$

$$(7.41)$$

This means that the permutation symbol is a relative tensor of weight -1

#### 7.3.6 The metric tensor

The metric tensor relates increments of distance to increments in the coordinates. The square of an increment of distance ds can be expressed in the displacement dx as:

$$ds^2 = dx \cdot dx \tag{7.42}$$

The displacement vector can be expressed using the chain rule as:

$$dx^{\alpha} = \frac{\partial x^{\alpha}}{\partial \xi^{\beta}} d\xi^{\beta} \tag{7.43}$$

$$ds^{2} = \sum_{\alpha=1}^{3} \frac{\partial x^{\alpha}}{\partial \xi^{\beta}} \frac{\partial x^{\alpha}}{\partial \xi^{\gamma}} d\xi^{\beta} d\xi^{\gamma} =$$

$$(7.44)$$

$$g_{\beta\gamma}d\xi^{\beta}d\xi^{\gamma}, \quad g_{\beta\gamma} = \sum_{\alpha=1}^{3} \frac{\partial x^{\alpha}}{\partial \xi^{\beta}} \frac{\partial x^{\alpha}}{\partial \xi^{\gamma}}$$

Close inspection of  $g_{\beta\gamma}$  reveals that it corresponds with the scalar product of the two covariant base vectors:  $\mathbf{a}_{(\beta)} \cdot \mathbf{a}_{(\gamma)}$ . It is easily shown that the metric tensor is indeed an absolute covariant tensor or order two:

$$\tilde{g}_{\alpha\beta} = \sum_{\gamma=1}^{3} \frac{\partial x^{\gamma}}{\partial \eta^{\alpha}} \frac{\partial x^{\gamma}}{\partial \eta^{\beta}} = \sum_{\gamma=1}^{3} \frac{\partial x^{\gamma}}{\partial \xi^{\mu}} \frac{\partial \xi^{\mu}}{\partial \eta^{\alpha}} \frac{\partial x^{\gamma}}{\partial \xi^{\sigma}} \frac{\partial \xi^{\sigma}}{\partial \eta^{\beta}} = \frac{\partial \xi^{\mu}}{\partial \eta^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial \eta^{\beta}} \sum_{\gamma=1}^{3} \frac{\partial x^{\gamma}}{\partial \xi^{\mu}} \frac{\partial x^{\gamma}}{\partial \xi^{\sigma}}$$
(7.45)

Obviously this tensor is symmetric, because the order of the terms is irrelevant:

$$g_{\alpha\beta} = g_{\beta\alpha} \tag{7.46}$$

Completely analogously a contravariant version and a mixed version of the metric tensor can be formed:

$$g^{\alpha\beta} = \boldsymbol{a}^{(\alpha)} \cdot \boldsymbol{a}^{(\beta)}$$

$$g^{\alpha}_{\beta} = \boldsymbol{a}^{(\alpha)} \cdot \boldsymbol{a}_{(\beta)} = \delta^{\alpha}_{\beta}$$
(7.47)

For the Cartesian case  $x = \xi$  we have

$$\frac{\partial x^{\alpha}}{\partial \xi^{\beta}} = \delta^{\alpha}_{\beta},\tag{7.48}$$

and this means that the metric tensor corresponds to the identity matrix:

$$g_{\alpha\alpha} = 1 = g^{\alpha\alpha} \text{(no summation)}, \qquad g_{\alpha\beta} = g^{\alpha\beta}$$
 (7.49)

Without specification of the index, q refers to the following determinant:

$$g = \left| \left( \boldsymbol{a}_{(1)} \boldsymbol{a}_{(2)} \boldsymbol{a}_{(3)} \right)^{T} \left( \boldsymbol{a}_{(1)} \boldsymbol{a}_{(2)} \boldsymbol{a}_{(3)} \right) \right| =$$

$$\left| \boldsymbol{a}_{(1)} \boldsymbol{a}_{(2)} \boldsymbol{a}_{(3)} \right|^{2}$$

$$(7.50)$$

That is why we refer to  $|a_{(1)}a_{(2)}a_{(3)}|$  as  $\sqrt{g}$ .

#### 7.3.7 Contraction

Contraction refers to the summation over a set of indices (a single super and subscript of course) to lower the order of a tensor by 2:

$$T_{\beta}^{\alpha\beta} = V^{\alpha} \tag{7.51}$$

$$T_{\beta}^{\alpha\beta} = V^{\alpha}$$
 (7.51)  
$$g^{\alpha\beta}T_{\beta\gamma} = S_{\gamma}^{\alpha}$$
 (7.52)

We can use contraction to raise and lower indices. The process is simple, the outcome is simple, but it is important to understand and keep in mind what you are actually doing: expressing the entities in a different coordinate system. Recall that the contravariant components of the velocity vector are given by:

$$U^{\beta} = \boldsymbol{a}^{(\beta)} \cdot \boldsymbol{u} = \boldsymbol{a}^{(\beta)} \cdot \left( \boldsymbol{a}^{(\alpha)} U_{\alpha} \right) = g^{\beta \alpha} U_{\alpha}, \tag{7.53}$$

where we used an expansion in the contravariant base vectors to express the vector u. Apparently, we can express the contravariant components of the velocity in the covariant components using the contravariant metric tensor Similarly we can express the covariant components in the contravariant components using the covariant metric tensor.

$$U_{\beta} = \boldsymbol{a}_{(\beta)} \cdot \boldsymbol{u} = \boldsymbol{a}_{(\beta)} \cdot \left(\boldsymbol{a}_{(\alpha)} U^{\alpha}\right) = g_{\alpha\beta} U^{\alpha} \tag{7.54}$$

The general rule to raise indices of a general tensor is given as follows:

$$g^{\beta_1\gamma_1}T^{\alpha_1\alpha_1\alpha_2\dots}_{\beta_1\beta_2\dots\beta_\xi} = T^{\gamma_1\alpha_1\alpha_2\dots}_{\beta_2\dots\beta_\xi}$$

$$(7.55)$$

Dummy indices can be lowered or raised as well:

$$T^{\alpha\beta}N_{\beta} = T^{\alpha}_{\beta}N^{\beta},\tag{7.56}$$

but of course  $T^{\alpha\beta} \neq T^{\alpha}_{\beta}$ .

## 7.3.8 The inner product

Depending on the basis of a pair of vectors the inner product can take a more or less compact form:

$$\boldsymbol{u} \cdot \boldsymbol{v} = (\boldsymbol{a}_{(\alpha)} U^{\alpha}) \cdot (\boldsymbol{a}^{(\beta)} V_{\beta}) = \delta_{\alpha}^{\beta} U^{\alpha} V_{\beta} \text{ (summation over 3 terms)}$$
 (7.57)

$$\boldsymbol{u} \cdot \boldsymbol{v} = (\boldsymbol{a}_{(\alpha)} U^{\alpha}) \cdot (\boldsymbol{a}_{(\beta)} V^{\beta}) = g_{\alpha\beta} U^{\alpha} V^{\beta} \text{ (summation over 9 terms)}$$
 (7.58)

## 7.3.9 Physical component

If n is some unit vector (length one, direction arbitrary) then the projection of a vector u on n (i.e.  $u \cdot n$ ) is called the physical component of u in the direction  $\boldsymbol{n}$ , and denoted by  $u_p(\boldsymbol{n})$ . We have

$$u_n(\mathbf{n}) = U^{\alpha} N_{\alpha} = U_{\alpha} N^{\alpha} \tag{7.59}$$

In a Cartesian system  $u_{\alpha}$  is the physical component in the  $x^{\alpha}$ -direction, but in general  $U_{\alpha}$  is not the physical component in the  $\xi^{\alpha}$ -direction. The unit vector parallel to  $a_{(\alpha)}$  for some  $\alpha$  is

$$\mathbf{n} = \mathbf{a}_{(\alpha)}/|\mathbf{a}_{(\alpha)}| = \mathbf{a}_{(\alpha)}/\sqrt{g_{\alpha\alpha}}$$
 (no summation) (7.60)

so that

$$N^{\beta} = g_{\alpha}^{\beta} / \sqrt{g_{\alpha\alpha}} \quad \text{(no summation)} \tag{7.61}$$

which gives

$$u_p(\mathbf{n}) = U_\alpha / \sqrt{g_{\alpha\alpha}}$$
 (no summation). (7.62)

Similarly, the covariant representation of the unit vector parallel to  $\boldsymbol{a}^{(\alpha)}$  for some  $\alpha$  is

$$N_{\beta} = \delta_{\beta}^{\alpha} / \sqrt{g^{\alpha \alpha}}$$
 (no summation) (7.63)

so that

$$u_p(\mathbf{n}) = U^{\alpha} / \sqrt{g^{\alpha \alpha}}$$
 (no summation) (7.64)

We will also need physical components of second order tensors, in order to evaluate the shear stress at a wall, for example. In Cartesian coordinates, in a fluid the stress component in the  $x^{\alpha}$ -direction on a surface element with unit normal n is given by

$$f^{\alpha} = t^{\alpha\beta} n_{\beta} \tag{7.65}$$

with  $t^{\alpha\beta}$  the (symmetric) stress tensor. The physical stress component in the direction of a unit vector m is given by

$$\mathbf{f} \cdot \mathbf{m} = m_{\alpha} t^{\alpha \beta} n_{\beta} \tag{7.66}$$

Because this is a tensor equation it holds in every coordinate system:

$$\boldsymbol{f} \cdot \boldsymbol{m} = M_{\alpha} T^{\alpha\beta} N_{\beta} \tag{7.67}$$

Suppose we want to determine the physical shear stress at a wall along which  $\xi^3 = constant$ . The unit normal to the surface is given by (7.61) with  $\alpha = 3$ . This gives

$$\mathbf{f} \cdot \mathbf{m} = M_{\alpha} T^{\alpha 3} / \sqrt{q^{33}} \tag{7.68}$$

For the component perpendicular to the wall  $M_{\alpha}$  is given by (7.63) with  $\alpha = 3$ , resulting in

$$\mathbf{f} \cdot \mathbf{m} = T^{33} / \sqrt{g^{33}} \tag{7.69}$$

## 7.3.10 Christoffel symbols

The distinguishing property of a curvilinear mapping is that the base vectors, as opposed to the Cartesian case are *not* constant, but depend on the local coordinates. Both their direction and magnitude are coordinate dependent, and spatial derivatives are nonzero. Taking the derivative of the covariant base vectors leads to:

$$\frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} = \frac{\partial}{\partial \xi^{\beta}} \left( \frac{\partial \boldsymbol{x}}{\partial \xi^{\alpha}} \right) = \frac{\partial^2 \boldsymbol{x}}{\partial \xi^{\beta} \partial \xi^{\alpha}} = \frac{\partial \boldsymbol{a}_{(\beta)}}{\partial \xi^{\alpha}}$$
(7.70)

The derivative is a vector quantity and can be expressed in either the covariant or contravariant base vectors in the following way, using coefficients to be defined:

$$\frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} = [\alpha \beta, \gamma] \boldsymbol{a}^{(\gamma)} \text{ (expressed in the contravariant basis vectors)}$$
 (7.71)

$$\frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} = \Gamma^{\gamma}_{\alpha\beta} \boldsymbol{a}_{(\gamma)} \text{ (expressed in the covariant basis vectors)}$$
 (7.72)

Taking the inner product with the base vector  $\mathbf{a}_{(\gamma)}$  of (7.71) gives:

$$\mathbf{a}_{(\gamma)} \frac{\partial \mathbf{a}_{(\alpha)}}{\partial \xi^{\beta}} = [\alpha \beta, \gamma],$$
 (7.73)

where  $[\alpha\beta, \gamma]$  denotes the *Christoffel symbol of the first kind*. Taking the inner product with the base vector  $\boldsymbol{a}^{(\gamma)}$  of (7.72) gives:

$$\mathbf{a}^{(\gamma)} \frac{\partial \mathbf{a}_{(\alpha)}}{\partial \xi^{\beta}} = \Gamma^{\gamma}_{\alpha\beta},\tag{7.74}$$

where  $\Gamma^{\gamma}_{\alpha\beta}$  denotes the *Christoffel symbol of the second kind*. Obviously, the two kinds of Christoffel symbols are closely related. We can contract the contravariant base vectors with the metric tensor to express these in the covariant base vectors.

$$\Gamma^{\gamma}_{\alpha\beta} = \boldsymbol{a}_{(\gamma)} \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} = \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} g^{\delta\gamma} \boldsymbol{a}_{(\delta)} = g^{\delta\gamma} [\alpha\beta, \delta] = g^{\gamma\delta} [\alpha\beta, \delta]. \tag{7.75}$$

Reversely we can contract the covariant base vectors with the metric tensor to express these in the contravariant base vectors:

$$[\alpha\beta, \gamma] = \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} \boldsymbol{a}_{(\gamma)} = \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} g_{\gamma\delta} \boldsymbol{a}^{(\delta)} = g_{\gamma\delta} \Gamma^{\delta}_{\alpha\beta}$$
(7.76)

Similarly, we can express the derivatives of the contravariant base vectors using Christoffel symbols. Taking derivatives of the relation (7.4) on the left and right hand side gives:

$$\frac{\partial}{\partial \xi^{\gamma}} \left( \boldsymbol{a}^{(\alpha)} \cdot \boldsymbol{a}_{(\beta)} \right) = \frac{\partial}{\partial \xi^{\gamma}} \delta^{\alpha}_{\beta} \Leftrightarrow \tag{7.77}$$

$$\boldsymbol{a}^{(\alpha)} \frac{\partial}{\partial \xi^{\gamma}} \boldsymbol{a}_{(\beta)} + \boldsymbol{a}_{(\beta)} \frac{\partial}{\partial \xi^{\gamma}} \boldsymbol{a}^{(\alpha)} \Leftrightarrow \tag{7.87}$$

$$\boldsymbol{a}_{(\beta)} \frac{\partial \boldsymbol{a}^{(\alpha)}}{\partial \xi^{\gamma}} = -\boldsymbol{a}^{(\alpha)} \frac{\partial}{\partial \xi^{\gamma}} \boldsymbol{a}_{(\beta)} = -\Gamma^{\alpha}_{\beta\gamma} \Leftrightarrow \frac{\partial \boldsymbol{a}_{(\beta)}}{\partial \xi^{\gamma}} = -\Gamma^{\alpha}_{\beta\gamma} \boldsymbol{a}^{(\beta)}$$

Of course, this derivative can be expressed in the covariant base vectors using the metric tensor. Important facts concerning the Christoffel symbols are:

- There are 18 Christoffel symbols in 3D and 6 in 2D (per kind that is).
- The Christoffel symbols contain second derivatives of the coordinate mapping  $x(\xi)$ :

$$\Gamma^{\gamma}_{\alpha\beta} = \boldsymbol{a}^{(\gamma)} \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\beta}} = \boldsymbol{a}^{(\gamma)} \frac{\partial^{2} \boldsymbol{x}}{\partial \xi^{\alpha} \partial \xi^{\beta}}$$
(7.78)

This means that these Christoffel symbols are only defined if the mapping  $x(\xi) \in C^2$ . If the coordinate mapping is defined discretely and the Christoffel symbols are approximated using finite differences, we can run into problems if the mapping is not sufficiently smooth. For this reason a discretisation can be formulated that avoids the use of the Christoffel symbols all together, see van Beek, van Nooyen, and Wesseling (1995).

- The Christoffel symbols are <u>not</u> tensors. This follows if we expand the Christoffel symbols in a different coordinate system:

$$\tilde{\Gamma}_{\alpha\beta}^{\gamma} = \frac{\partial \tilde{a}_{(\alpha)}}{\partial \tilde{\xi}^{\beta}} \tilde{a}^{(\gamma)} = \frac{\partial}{\partial \tilde{\xi}} \left( \frac{\partial \xi^{\delta}}{\partial \tilde{\xi}^{\alpha}} a_{(\delta)} \right) \frac{\tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} a^{(\mu)} =$$

$$\frac{\partial^{2} \xi^{\delta}}{\partial \tilde{\xi}^{\beta} \partial \tilde{\xi}^{\alpha}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} a_{(\delta)} a^{(\mu)} + \frac{\partial \xi^{\delta}}{\partial \tilde{\xi}^{\alpha}} \frac{\partial a_{(\delta)}}{\partial \xi^{\sigma}} \frac{\partial \xi^{\sigma}}{\partial \tilde{\xi}^{\beta}} \frac{\tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} a^{(\mu)} =$$

$$\frac{\partial^{2} \xi^{\delta}}{\partial \tilde{\xi}^{\beta} \partial \tilde{\xi}^{\alpha}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} \delta^{\mu}_{\sigma} + \frac{\partial \xi^{\delta}}{\partial \tilde{\xi}^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial \tilde{\xi}^{\beta}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} \Gamma^{\mu}_{\delta\sigma}$$

$$\frac{\partial^{2} \xi^{\delta}}{\partial \tilde{\xi}^{\beta} \partial \tilde{\xi}^{\alpha}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\sigma}} + \frac{\partial \xi^{\delta}}{\partial \tilde{\xi}^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial \tilde{\xi}^{\beta}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} \Gamma^{\mu}_{\delta\sigma}$$

$$\frac{\partial^{2} \xi^{\delta}}{\partial \tilde{\xi}^{\beta} \partial \tilde{\xi}^{\alpha}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\sigma}} + \frac{\partial \xi^{\delta}}{\partial \tilde{\xi}^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial \tilde{\xi}^{\beta}} \frac{\partial \tilde{\xi}^{\gamma}}{\partial \xi^{\mu}} \Gamma^{\mu}_{\delta\sigma}$$

Apparently, transformation to the tilded system introduces an additional term, next to the expected mixed third order transformation if the Christoffel symbol would have been a tensor.

#### 7.3.11 Differentiating the metric tensor

Because the components of the metric tensor are just the inner products of the vectors in the covariant or contravariant bases we can use the Christoffel symbols to express the derivative of the metric tensor:

$$\frac{\partial g_{\alpha\beta}}{\partial \xi^{\gamma}} = \frac{\partial}{\partial \xi^{\gamma}} \left( \boldsymbol{a}_{(\alpha)} \cdot \boldsymbol{a}_{(\beta)} \right) = \boldsymbol{a}_{(\alpha)} \frac{\partial \boldsymbol{a}_{(\beta)}}{\partial \xi^{\gamma}} + \boldsymbol{a}_{(\beta)} \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\gamma}} 
= \boldsymbol{a}_{(\alpha)} [\beta \gamma, \delta] \boldsymbol{a}^{(\delta)} + \boldsymbol{a}_{(\beta)} [\alpha \gamma, \delta] \boldsymbol{a}^{(\delta)} 
= [\beta \gamma, \delta] \delta_{\alpha}^{\delta} + [\alpha \gamma, \delta] \delta_{\beta}^{\delta} 
= [\beta \gamma, \alpha] + [\alpha \gamma, \beta]$$
(7.80)

We can use (7.80) to express the Christoffel symbols of the first and second kind in derivatives of the metric tensor:

$$[\alpha\beta, \gamma] = \frac{1}{2} \left( \frac{\partial g_{\beta\gamma}}{\partial \xi^{\alpha}} + \frac{\partial g_{\alpha\gamma}}{\partial \xi^{\beta}} - \frac{\partial g_{\alpha\beta}}{\partial \xi^{\gamma}} \right)$$
(7.81)

$$\Gamma^{\gamma}_{\alpha\beta} = \frac{1}{2}g^{\gamma\delta} \left( \frac{\partial g_{\beta\delta}}{\partial \xi^{\alpha}} + \frac{\partial g_{\alpha\delta}}{\partial \xi^{\beta}} - \frac{\partial g_{\alpha\beta}}{\partial \xi^{\delta}} \right)$$
 (7.82)

We can now contract over 1 subscript and one superscript:

$$\Gamma^{\alpha}_{\alpha\beta} = \frac{1}{2} g^{\alpha\delta} \frac{\partial g_{\alpha\delta}}{\partial \xi^{\beta}} \tag{7.83}$$

$$\frac{\partial g}{\partial g_{\alpha\delta}} = G^{\alpha\delta} = gg^{\alpha\delta} \tag{7.84}$$

$$\frac{\partial g}{\partial \xi^{\beta}} = \frac{\partial g}{\partial g_{\alpha\delta}} \frac{\partial g_{\alpha\delta}}{\partial \xi^{\beta}} = gg^{\alpha\delta} \frac{\partial g_{\alpha\delta}}{\partial \xi^{\beta}}$$
 (7.85)

$$\Gamma^{\alpha}_{\alpha\beta} = \frac{1}{2} \frac{1}{g} \frac{\partial g}{\partial \xi^{\beta}} = \frac{1}{2} \frac{\partial}{\partial \xi^{\beta}} (\ln g) = \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\beta}} (\sqrt{g})$$
 (7.86)

This expression will be used shortly to formulate the divergence operator in general coordinates.

## 7.3.12 Differentiating a vector field

Taking the derivative of a vector field in the Cartesian case, means taking the derivatives of the field components, because of the fact that the base vectors are independent of the spatial coordinates. We have just seen that in the curvilinear case the base vectors do depend on the spatial coordinates, and we have to apply the product rule in the following way:

$$\frac{\partial}{\partial \xi^{\gamma}} \boldsymbol{u} = \frac{\partial}{\partial \xi^{\gamma}} \left( \boldsymbol{a}_{(\alpha)} U^{\alpha} \right) = U^{\alpha} \frac{\partial \boldsymbol{a}_{(\alpha)}}{\partial \xi^{\gamma}} + \frac{\partial U^{\alpha}}{\partial \xi^{\gamma}} \boldsymbol{a}_{(\alpha)}$$

$$= U^{\alpha} \Gamma^{\beta}_{\alpha \gamma} \boldsymbol{a}_{(\beta)} + \frac{\partial U^{\alpha}}{\partial \xi^{\gamma}} \boldsymbol{a}_{(\alpha)}$$

$$= U^{\delta} \Gamma^{\alpha}_{\delta \gamma} \boldsymbol{a}_{(\alpha)} + \frac{\partial U^{\alpha}}{\partial \xi^{\gamma}} \boldsymbol{a}_{(\alpha)}$$

$$= \left( U^{\delta} \Gamma^{\alpha}_{\delta \gamma} + \frac{\partial U^{\alpha}}{\partial \xi^{\gamma}} \right) \boldsymbol{a}_{(\alpha)}$$

$$= U^{\alpha}_{\beta} \boldsymbol{a}_{(\alpha)},$$
(7.87)

where

$$U^{\alpha}_{,\gamma} := \left( U^{\delta} \Gamma^{\alpha}_{\delta\gamma} + \frac{\partial U^{\alpha}}{\partial \xi^{\gamma}} \right) \tag{7.88}$$

is the covariant derivative of the contravariant vector  $U^{\alpha}$ . We can also start from a representation of the vector  $\boldsymbol{u}$  in the contravariant basis. This leads similarly to:

$$\frac{\partial \boldsymbol{u}}{\partial \xi^{\gamma}} = U_{\alpha,\gamma} \boldsymbol{a}^{(\alpha)}, \tag{7.89}$$

where

$$U_{\alpha,\gamma} := \frac{\partial U_{\alpha}}{\partial \mathcal{E}^{\gamma}} - \Gamma^{\delta}_{\alpha\gamma} U_{\delta} \tag{7.90}$$

So we have the following representations for the covariant derivative

$$\frac{\partial \boldsymbol{u}}{\partial \xi^{\gamma}} = U_{,\gamma}^{\alpha} \boldsymbol{a}_{(\alpha)} = U_{\alpha,\gamma} \boldsymbol{a}^{(\alpha)} = U_{\alpha,\gamma} g^{\alpha\beta} \boldsymbol{a}_{(\beta)} = U_{\beta,\gamma} g^{\beta\alpha} \boldsymbol{a}_{(\alpha)}$$
(7.91)

The covariant derivative is the coordinate-invariant form of the derivative. Note that in the Cartesian case, where all Christoffel symbols vanish, it corresponds to the 'regular' derivative. Also note that the complexity of the term is actually hidden.

## 7.3.13 Formulating equations in coordinate invariant form

With all building blocks defined, we are now able to formulate equations in coordinate invariant form. The basic rule is that if an equation is a proper tensor equation, it can be brought in coordinate invariant form. What defines a proper tensor equation? Well, first all entries in the equations should of course be tensors (regarding vectors as tensors of order 1, and scalars as tensors of order 0), and transform according to either the covariant or contravariant transformation rules. Then it should hold that any free index in the equation should occur in a similar way in each term of the equation. So if in one of the terms  $\alpha$  occurs as a free subscript, it can not occur in other terms as a free superscript. Finally, recurring indices (indicating contraction) should occur as a matched set of super and subscript.

If an equation qualifies as a proper tensor equation, then by *Ricci's Lemma* it holds in any coordinate system, when all derivatives in the equation are indeed regarded as covariant derivatives. We will now show how to formulate a number of operators and equations in coordinate invariant form.

## 7.3.14 Divergence operator

In Cartesian coordinates (remember, base vectors are constant) in 2D, the divergence operator is given by

$$\nabla \cdot \boldsymbol{u} = u_{\alpha,\alpha} \tag{7.92}$$

This is not a proper tensor equation, because the repeated index  $\alpha$  does not occur as a matched pair of a subscript and superscript. Because in the Cartesian case there is no difference between the covariant and the contravariant components, we can rewrite (7.92) as:

$$\nabla \cdot \boldsymbol{u} = u^{\alpha}_{\ \alpha} \tag{7.93}$$

Now (7.93) does qualify as a proper tensor equation and according to Ricci's lemma it holds in any coordinate system, if we regard the derivatives as covariant derivatives. So, in general coordinates the divergence operator is given by:

$$\nabla \cdot \boldsymbol{u} = U^{\alpha}_{\alpha} \tag{7.94}$$

We will now derive a compact expression for the divergence operator in general coordinates that can be used in the finite volume discretisation of the Navier-Stokes equations.

$$\frac{\partial \boldsymbol{u}}{\partial \xi^{\alpha}} = U^{\beta}_{,\alpha} \boldsymbol{a}_{(\beta)} \tag{7.95}$$

If we multiply (7.95) by  $a^{(\alpha)}$ . This gives:

$$\frac{\partial \boldsymbol{u}}{\partial \xi^{\alpha}} \boldsymbol{a}^{(\alpha)} = U_{,\alpha}^{\beta} \boldsymbol{a}_{(\beta)} \boldsymbol{a}^{(\alpha)} = U_{,\alpha}^{\beta} \delta_{\beta}^{\alpha} = U_{,\alpha}^{\alpha} = \boldsymbol{\nabla} \cdot \boldsymbol{u}$$
 (7.96)

$$\mathbf{a}^{(\alpha)} \frac{\partial \mathbf{u}}{\partial \xi^{\alpha}} = \mathbf{a}^{(\alpha)} \frac{\partial}{\partial \xi^{\alpha}} \left( U^{\beta} \mathbf{a}_{(\beta)} \right), \tag{7.97}$$

$$= \mathbf{a}^{(\alpha)} \left( \frac{\partial U^{\beta}}{\partial \xi^{\alpha}} \mathbf{a}_{(\beta)} + U^{\beta} \frac{\partial \mathbf{a}_{(\beta)}}{\partial \xi^{\alpha}} \right),$$

$$= \mathbf{a}^{(\alpha)} \left( \frac{\partial U^{\beta}}{\partial \xi^{\alpha}} \mathbf{a}_{(\beta)} + U^{\beta} \Gamma_{\beta\alpha}^{\gamma} \mathbf{a}_{(\gamma)} \right),$$

$$= \frac{\partial U^{\beta}}{\partial \xi^{\alpha}} \delta^{\alpha}_{\beta} + U^{\beta} \Gamma^{\alpha}_{\beta\alpha} \delta^{\alpha}_{\gamma},$$

$$= \frac{\partial U^{\beta}}{\partial \xi^{\beta}} + U^{\beta} \Gamma^{\alpha}_{\beta\alpha},$$

$$= \frac{\partial U^{\alpha}}{\partial \xi^{\alpha}} + U^{\beta} \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\beta}} (\sqrt{g}),$$

$$= \frac{\partial U^{\alpha}}{\partial \xi^{\alpha}} + U^{\alpha} \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\alpha}} (\sqrt{g}),$$

$$= \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\alpha}} (\sqrt{g} U^{\alpha}).$$

So, in brief:

$$\nabla \cdot \boldsymbol{u} = \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\alpha}} \left( \sqrt{g} U^{\alpha} \right). \tag{7.98}$$

#### 7.3.15 Laplace operator

The Laplace operator applied to a scalar field  $\varphi$  in the Cartesian case is given by:

$$\nabla^2 \varphi = \nabla \cdot \nabla \varphi = (\varphi_{,\alpha})_{\alpha}. \tag{7.99}$$

Again, this equation (7.99) does not qualify as a tensor equation, because the repeated subscript  $\alpha$  does not occur as a matched pair of one subscript and one superscript. The problem is that the gradient operator is covariant:

$$\frac{\partial \varphi}{\partial \xi^{\alpha}} = \frac{\partial \varphi}{\partial x^{\beta}} \frac{\partial x^{\beta}}{\partial \xi^{\alpha}}.$$
 (7.100)

To make (7.99) a proper tensor equation we need to express the gradient vector in its contravariant components, so we contract with the contravariant metric tensor:

$$(\nabla \varphi)^{\alpha} = g^{\alpha\beta} \varphi_{,\beta}. \tag{7.101}$$

So the proper tensor form of the Cartesian Laplace operator is given by:

$$\nabla^2 \varphi = \left( g^{\alpha\beta} \varphi_{,\beta} \right)_{,\alpha}. \tag{7.102}$$

According to Ricci's lemma this tensor equation holds in any coordinate system if we regard all derivatives of vector quantities (like in the divergence operator) as covariant derivatives. Combining (7.98) and (7.101) leads to a very elegant expression for the Laplacian:

$$\nabla^2 \varphi = \frac{1}{\sqrt{g}} \frac{\partial}{\partial \xi^{\alpha}} \left( \sqrt{g} g^{\alpha \beta} \frac{\partial \varphi}{\partial x^{\beta}} \right)$$
 (7.103)

## 7.3.16 An example computation

As an illustration we will compute all the Christoffel symbols of the second kind for a cylindrical coordinate system  $x(r, \theta, z)$ :

$$x_1 = r\cos\theta \tag{7.104}$$

$$x_2 = r\sin\theta \tag{7.105}$$

$$x_3 = z \tag{7.106}$$

It turns out that are are only three nonzero (two unique) Christoffel symbols of the second kind:  $\Gamma^{\theta}_{\theta r}$ ,  $\Gamma^{\theta}_{r\theta}$  and  $\Gamma^{r}_{\theta \theta}$ 

The covariant base vectors are straightforward to find. (Check this!)

$$\mathbf{a}_{(r)} = (\cos \theta, \sin \theta, 0) \tag{7.107}$$

$$\mathbf{a}_{(\theta)} = (-r\sin\theta, r\cos\theta, 0) \tag{7.108}$$

$$\mathbf{a}_{(z)} = (0, 0, 1) \tag{7.109}$$

(7.110)

We also need the contravariant base vectors  $\mathbf{a}^{(r)}$  and  $\mathbf{a}^{(\theta)}$ :

$$\boldsymbol{a}^{(r)} = \frac{1}{\sqrt{g}} \left( \boldsymbol{a}_{(\theta)} \times \boldsymbol{a}_{(z)} \right) = \frac{1}{r} \begin{vmatrix} \boldsymbol{i} & \boldsymbol{j} & \boldsymbol{k} \\ -r \sin \theta & r \cos \theta & 0 \\ 0 & 0 & 1 \end{vmatrix}$$
 (7.111)

$$\boldsymbol{a}^{(\theta)} = \frac{1}{\sqrt{g}} \left( \boldsymbol{a}_{(z)} \times \boldsymbol{a}_{(r)} \right) = \frac{1}{r} \begin{vmatrix} \boldsymbol{i} & \boldsymbol{j} & \boldsymbol{k} \\ 0 & 0 & 1 \\ \cos \theta & \sin \theta & 0 \end{vmatrix}$$
 (7.112)

The two unique Christoffel symbols are then given by:

$$\Gamma_{\theta r}^{\theta} = \mathbf{a}^{(\theta)} \cdot \frac{\partial \mathbf{a}_{(\theta)}}{\partial r} = \frac{1}{2} \left( -\sin\theta, \cos\theta, 0 \right) \cdot \left( -\sin\theta, \cos\theta, 0 \right) = \frac{1}{r}$$
 (7.113)

$$\Gamma_{\theta\theta}^{r} = \boldsymbol{a}^{(r)} \cdot \frac{\partial \boldsymbol{a}_{(\theta)}}{\partial \theta} = (\cos \theta, \sin \theta, 0) \cdot (-r \cos \theta, -r \sin \theta, 0) = -r \qquad (7.114)$$

## 7.3.17 Navier-Stokes equations

We will now formulate the incompressible Navier-Stokes equations in coordinate invariant form, starting from the Navier-Stokes equations in Cartesian coordinates.

$$u^{\alpha}_{,\alpha} = 0 \tag{7.115}$$

$$\frac{\partial \rho u^{\alpha}}{\partial t} + \left(u^{\alpha} u^{\beta}\right)_{,\beta} = -p_{,\alpha} + \sigma_{,\beta}^{\alpha\beta}$$

$$\sigma^{\alpha\beta} = Re^{-1} \left(u_{,\beta}^{\alpha} + u_{,\alpha}^{\beta}\right)$$
(7.116)

$$\sigma^{\alpha\beta} = Re^{-1} \left( u^{\alpha}_{.\beta} + u^{\beta}_{.\alpha} \right) \tag{7.117}$$

To be able to cast this system of equations in coordinate invariant form, we need to verify if it qualifies as a proper set of tensor equations. Close inspection shows that the only problem occurs in the pressure gradient in the momentum conservation equation. In this term the free index  $\alpha$  appears as a subscript, while the other terms include this free index as a superscript.

This can be easily fixed by raising the subscript in the pressure gradient by multiplication with the contravariant metric tensor  $g^{\alpha\beta}$ . This leads to the following coordinate invariant form of the Navier-Stokes equations:

$$u^{\alpha}_{,\alpha} = 0 \tag{7.118}$$

$$u_{,\alpha}^{\alpha} = 0 \tag{7.118}$$

$$\frac{\partial \rho u^{\alpha}}{\partial t} + (u^{\alpha} u^{\beta})_{,\beta} = -(g^{\alpha\beta} p)_{,\beta} + \sigma_{,\beta}^{\alpha\beta} \tag{7.119}$$

$$\sigma^{\alpha\beta} = Re^{-1} \left( u^{\alpha}_{,\beta} + u^{\beta}_{,\alpha} \right) \tag{7.120}$$

Now we can transform (7.118) to coordinate invariant form, by simply replacing the Cartesian velocity components by the contravariant components and all derivatives of vector quantities by covariant derivatives:

$$U^{\alpha}_{,\alpha} = 0 \tag{7.121}$$

$$\frac{\partial \rho U^{\alpha}}{\partial t} + \left(U^{\alpha} U^{\beta}\right)_{,\beta} = -\left(g^{\alpha\beta} p\right)_{,\beta} + \sigma^{\alpha\beta}_{,\beta}$$

$$\sigma^{\alpha\beta} = Re^{-1} \left(U^{\alpha}_{,\beta} + U^{\beta}_{,\alpha}\right)$$
(7.122)

$$\sigma^{\alpha\beta} = Re^{-1} \left( U^{\alpha}_{.\beta} + U^{\beta}_{.\alpha} \right) \tag{7.123}$$

Exercise 7.3.2. If the mapping is not available in closed form, but only as a grid function  $x_{ij}(\xi_{ij})$  the contravariant basis vectors have to be approximated numerically. Derive a second order accurate approximation to the first and second contravariant basis vectors.

Exercise 7.3.3. Express the boundary condition:

$$\nabla \varphi(\mathbf{x}) \cdot \hat{\mathbf{n}} = \mathbf{g}(\mathbf{x}), \quad \mathbf{x} \in \Gamma, \tag{7.124}$$

where  $\Gamma$  corresponds to a  $\xi_1 = c$  curve in  $\varphi(\boldsymbol{\xi})$ , its derivatives with respect to  $\xi_{\alpha}$ and the geometrical quantities  $a^{(\alpha)}$ ,  $a_{(\beta)}$  and the contra- and covariant metric tensors.

#### Some self-test questions

How many grid points are in the stencil for the discretisation of (7.103)?

## 8. Introduction to classical grid generation

In this course we will touch upon the subject of grid generation, to construct structured and the unstructured grids in 2D. Both algebraic and differential methods for the generation of structured grids will be discussed, as well as two different approaches for the generation of unstructured triangulations. The techniques do certainly not reflect the state-of-the-art, but have proven their value and are still used today. The material is heavily based on Farrashkhalvat and Miles (2003), which should be consulted for a more elaborate discussion of the techniques presented. Knupp and Steinberg (1993) provides a mathematical background for structured grid generation techniques, and contains sample programs.

Do not waste your time on building grid generation software for your own project. Many excellent grid generation programs are freely available. However, it is essential to understand the basics of these methods to appreciate their capabilities and computational cost.

## 8.1 Starting point

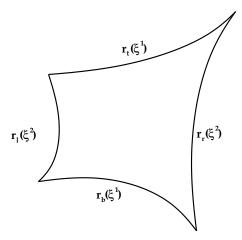
The starting point for grid generation is a description of the boundary of the domain as a set of parameterized curves. The generation of these curves is the field of Computer Aided Design, and outside the scope of this discussion. The *standard* parameterization is done in the normalized arc length. For structured grid generation the domain has to have the topology of a square: It should have four vertices and four edges.

Because we plan to map this region to a unit square, the parameterization of the two opposing pairs of edges is already performed in the appropriate parameters, as illustrated in Fig. 8.1.

Complex regions that do not have the topology of a unit square can be decomposed into a set of non overlapping subdomains (so-called *blocks*, irrespective of the dimensionality of the geometry) that do have the 'right' topology, this leads to *multi-block* discretisation. Furthermore, we require that the geometry is not degenerate. This means that a pair of vertices should not coincide, and neither should one of the included angles of the four corners approach  $\pi$ . The grid will consists of the intersections of two sets of coordinate lines  $\xi^{\alpha}$  in the following way:

$$0 \le \xi_i^{\alpha} = \frac{i-1}{n^{\alpha}-1} \le 1, \quad i = 1, 2, \dots, n^{\alpha},$$
 (8.1)

and explicitly formulating an onto (bijective)  $C^1$  mapping with a positive Jacobian:



**Figure 8.1.** The starting point for the grid generation is a description of the boundary of the domain in four parameterized curves.

$$(x^1, x^2) = \mathbf{r}(\xi^1, \xi^2) \tag{8.2}$$

## 8.2 Structured grid generation

The aim of structured grid generation is the construction of an ordered set of boundary conforming rectilinear quadrilateral elements. A hierarchy of methods

- Algebraic methods
  - Coordinate transformation
  - Transfinite interpolation
- Differential methods
  - non-elliptic methods
  - elliptic methods

The properties of algebraic methods in a nutshell are

- Algebraic methods are fast and cheap to apply.
- Algebraic methods work fine for convex regions,
- Algebraic methods can be used to generate a starting grid for differential methods, to further improve grid smoothness or (local) orthogonality.
- Algebraic methods allow direct control of the grid density.
- Algebraic methods propagate geometric singularities/non-smoothness (e.g. a kink) of the boundary into the interior of the computational domain.

On the other hand,

- Differential methods are relatively time-consuming/expensive to apply.
- Differential methods require the inversion of a linear system,
- Differential methods produce intrinsically smooth grids, even for non-smooth boundaries.
- Differential methods can control local grid density (and orthogonality) but not necessarily in a transparent way.

## 8.2.1 Algebraic grid generation techniques

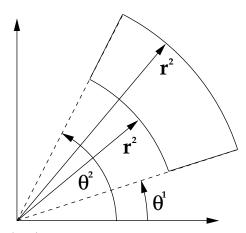
Naturally boundary fitting coordinate systems. The most straightforward, but least broadly applicable approach to generation of a boundary conforming grid is the use of analytic coordinate transformations, e.g. polar coordinates in 2D and cylindrical and spherical coordinates in 3D. In the rare case the computational domain corresponds to a circle segment polar coordinates can transform a circular wedge bounded by the curves to a rectangular region: Next, and additional scaling can be applied that maps the unit square  $\Omega: \{(\xi^1, \xi^2) \in [0, 1] \times [0, 1]\}$  to the rectangular region  $S: \{(\theta, r)|\theta^1 \leq \theta \leq \theta^2, R^1 \leq r \leq R^2\}$ 

$$\begin{array}{ll} \theta(\xi^1, \xi^2) \ = \ (1 - \xi^1)\theta^1 + \xi^1\theta^2, \quad 0 \le \xi^1, \xi^2 \le 1 \\ r(\xi^1, \xi^2) \ = \ (1 - \xi^2)R^1 + \xi^2R^2, \quad 0 \le \xi^1, \xi^2 \le 1 \end{array} \tag{8.3}$$

The combined transformation now reads:

$$x^{1}(\xi^{1}, \xi^{2}) = \left[ (1 - \xi^{2})R^{1} + \xi^{2}R^{2} \right] \cos((1 - \xi^{1})\theta^{1} + \xi^{1}\theta^{2})$$

$$x^{2}(\xi^{1}, \xi^{2}) = \left[ (1 - \xi^{2})R^{1} + \xi^{2}R^{2} \right] \sin((1 - \xi^{1})\theta^{1} + \xi^{1}\theta^{2})$$
(8.4)



**Figure 8.2.** In the (rare) case the computational domain coincides with a circular wedge, a transformation based on scaled polar coordinates can be defined to define the mapping  $\mathbf{r}_0(\xi^1, \xi^2)$ .

Unidirectional interpolation. If the domain has the topology of a square, with one pair of opposing straight sides (Fig. 8.3) and one pair of opposing curved sides, we can generate a grid using linear interpolation between the opposing curves. First the two curves  $\mathbf{r}_1$  and  $\mathbf{r}_2$  are parameterized in the following way:

$$\mathbf{r}_1 = \mathbf{r}_1(\xi^2), \quad 0 \le \xi^2 \le 1,$$
  
 $\mathbf{r}_2 = \mathbf{r}_2(\xi^2), \quad 0 \le \xi^2 \le 1.$ 
(8.5)

To allow for more control in the shape and distribution of the grid lines, additional *control* curves can be added in the interior. These control curves are

parameterized in the same parameters as the boundary curves. The interpolation connects points on the curves (boundary and control) with equal values of  $\xi^2$ , linearly interpolating across the interior of the domain. The most straightforward choice for the interpolation polynomials are the Lagrange interpolation polynomials. When interpolating between a set of points  $(x_i^1, x_i^2)$ , the interpolants are defined as:

$$L_i(x^1) = \prod_{j=0, \ j \neq i}^n \frac{x^1 - x_j^1}{x_i^1 - x_j^1}$$
(8.6)

which have the property:

$$L_i(x_i^1) = \delta_{ij} \tag{8.7}$$

The polynomial that interpolates between the points  $(x_i^1, x_i^2)$  is given by:

$$p(x^{1}) = \sum_{i=0}^{n} L_{i}(x^{1})x_{i}^{2}$$
(8.8)

The most simple interpolation uses only first order interpolation:

$$L_0(x^1) = \frac{x^1 - x_1^1}{x_0^1 - x_1^1} \qquad L_1(x^1) = \frac{x^1 - x_0^1}{x_1^1 - x_0^1}$$
(8.9)

We will use a normalized variable  $\xi^1$  such that:

$$x^{1} = (x_{1}^{1} - x_{0}^{1})\xi^{1}, \quad 0 \le \xi^{1} \le 1$$
 (8.10)

The interpolation polynomials now take the simple form:

$$L_0(\xi^1) = \frac{\xi^1 - \xi_1^1}{\xi_0^1 - \xi_1^1} = \frac{\xi^1 - 1}{0 - 1} = 1 - \xi^1$$

$$L_1(\xi^1) = \frac{\xi^1 - \xi_1^0}{\xi_1^1 - \xi_0^1} = \frac{\xi^1 - 0}{1 - 0} = \xi^1$$

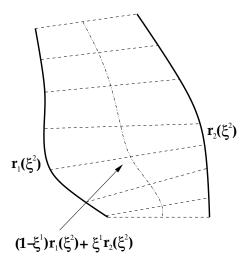
$$p(\xi^1) = x_0^2 (1 - \xi^1) + x_1^2 \xi^1$$
(8.11)

Application of this interpolation polynomial to interpolate between two points on the boundary curves that have the same value of the parameter  $\xi^2$  defines a mapping in the whole domain in the following way:

$$\mathbf{r}(\xi^1, \xi^2) = (1 - \xi^1)\mathbf{r}_0(\xi^2) + \xi^1\mathbf{r}_1(\xi^2), \quad 0 \le \xi^1, \xi^2 \le 1$$
 (8.12)

With the geometry of the two curves fixed, the only way to control the clustering of grid cells is by re-parameterization of the curves. To be able to allow for clustering of grid cells, such as is required to have efficiency and accuracy uniform in the Peclet or Reynolds number, as was demonstrated in the Cartesian case in Section \*\*.

$$\mathbf{r}(\xi^1, \xi^2) = \mathbf{r}_0(\xi^2) L_0(\xi^1) + \mathbf{r}_{\frac{1}{2}}(\xi^2) L_{\frac{1}{2}}(\xi^2) + \mathbf{r}_1(\xi^2) L_1(\xi^1)$$
(8.13)



**Figure 8.3.** When the domain has the topology of a square with one set of opposing straight sides, and a set of opposing curved sides, unidirectional interpolation can be used to define a mapping  $\mathbf{r}_0(\xi^1, \xi^2)$ .

$$L_0(\xi^1) \equiv \frac{(\xi^1 - \xi_{\frac{1}{2}}^1)(\xi^1 - \xi_{\frac{1}{2}}^1)}{(\xi_0^1 - \xi_{\frac{1}{2}}^1)(\xi_0^1 - \xi_{\frac{1}{2}}^1)} = 2(\xi^1 - \frac{1}{2})(\xi^1 - 1)$$
(8.14)

$$L_{\frac{1}{2}}(\xi^1) \equiv \frac{(\xi^1 - \xi_0^1)(\xi^1 - \xi_1^1)}{(\xi_{frac12}^1 - \xi_0^1)(\xi_{\frac{1}{2}}^1 - \xi_1^1)} = 4\xi^1(1 - \xi^1)$$
(8.15)

$$L_1(\xi^1) \equiv \frac{(\xi^1 - \xi_0^1)(\xi^1 - \xi_{\frac{1}{2}}^1)}{(\xi_1^1 - \xi_0^1)(\xi_1^1 - \xi_{\frac{1}{2}}^1)} = \xi^1(2\xi^1 - 1)$$
(8.16)

This approach allows us to control grid density in the interior of the domain, but not the *orthogonality* of the grid in the vicinity of the boundary. Making the grid lines orthogonal to the boundary will improve the local accuracy of the discretisation and is also needed in a multi-block environment to guarantee smoothness of the grid across the block boundaries. Controlling the direction of the grid lines is accomplished by switching to Hermite interpolation polynomials:

$$p(x^{1}) = \sum_{i=0}^{n} x_{i}^{2} H_{i}(x^{1}) + \sum_{i=0}^{n} (x^{2})_{i}' \tilde{H}_{i}(x^{1})$$
(8.17)

**Transfinite interpolation.** Suppose our domain is bounded by four curves, and we have a mapping available that maps the unit square in the curvilinear coordinate onto our domain in the way depicted in Figure.\*\* Define the following projection  $P_{\xi^1}$  defined by the unidirectional interpolation between the left and right curve of the domain:

$$\tilde{\mathbf{r}}_0(\xi^1, \xi^2) = P_{\xi^1}(\mathbf{r}_0(\xi^1, \xi^2)) = (1 - \xi^1)\mathbf{r}_0(0, \xi^2) + \xi^1\mathbf{r}_0(1, \xi^2)$$
(8.18)

This projection is an approximation of the original map  $\mathbf{r}_0(\xi^1, \xi^2)$ : it leaves the right and left boundary intact, but replaces the original lower and upper boundary of the domain by straight lines. Similarly, a projection  $P_{\xi^2}$  can be defined that leaves the upper and lower boundary intact, but replaces the left and right hand side by straight lines:

$$\tilde{\mathbf{r}}_0(\xi^1, \xi^2) = P_{\xi^2}(\mathbf{r}_0(\xi^1, \xi^2)) = (1 - \xi^2)\mathbf{r}_0(\xi^1, 0) + \xi^1\mathbf{r}_0(\xi^1, 1)$$
(8.19)

It is easily shown that both  $P_{\xi^1}$  and  $P_{\xi^2}$  are projections (idempotent operations), because repeated application of for example  $P_{\xi^1}$  gives:

$$\tilde{\mathbf{r}}_{0}(\xi^{1}, \xi^{2}) = P_{\xi^{1}}(P_{\xi^{1}}(\mathbf{r}_{0}(\xi^{1}, \xi^{2}))) = (8.20)$$

$$(1 - \xi^{1}) \left[ (1 - 0)\mathbf{r}_{0}(0, \xi^{2}) + 0 \cdot \mathbf{r}_{0}(1, \xi^{2}) \right] + \xi^{1} \left[ (1 - 1)\mathbf{r}_{0}(0, \xi^{2}) + 1 \cdot \mathbf{r}_{0}(1, \xi^{2}) \right] = (1 - \xi^{1})\mathbf{r}_{0}(0, \xi^{2}) + \xi^{1}\mathbf{r}_{0}(1, \xi^{2}) = P_{\xi^{1}}(\mathbf{r}_{0}(\xi^{1}, \xi^{2}))$$

Combining the two projections leads to a mapping that replaces all four sides by straight lines, leaving only the four vertices of the patch unaffected:

$$\tilde{\mathbf{r}}_{0}(\xi^{1}, \xi^{2}) = P_{\xi^{1}}(P_{\xi^{2}}(\mathbf{r}_{0}(\xi^{1}, \xi^{2}))) = (8.21)$$

$$(1 - \xi^{1})(1 - \xi^{2})\mathbf{r}_{0}(0, 0) + (1 - \xi^{1})\xi^{2}\mathbf{r}_{0}(0, 1) + \xi^{1}(1 - \xi^{2})\mathbf{r}_{0}(1, 0) + \xi^{1}\xi^{2}\mathbf{r}_{0}(1, 1)$$

However, what we want to do is somehow use the projection operators to extend the mapping from the <u>boundary</u> to the interior of the domain, but in such a way it leaves <u>all</u> four boundary curves unaffected. This disqualifies the use of unidirectional interpolation, because this replaced opposite edges by straight lines. However, it turns out that when the two projection operators are applied in the following way to the mapping:

$$(P_{\xi^1} + P_{\xi^1} - P_{\xi^1} P_{\xi^1}) (\mathbf{r}_0(\xi^1, \xi^2)) \equiv (P_{\xi^1} \oplus P_{\xi^2}) (\mathbf{r}_0), \tag{8.22}$$

where  $(P_{\xi^1} \oplus P_{\xi^2})$  denotes the *Boolean sum* of the two projections  $P_{\xi^1}$  and  $P_{\xi^2}$ . This mapping is given by:

$$\tilde{\mathbf{r}}_{0}(\xi^{1}, \xi^{2}) = (P_{\xi^{1}} \oplus P_{\xi^{2}}) (\mathbf{r}_{0}(\xi^{1}, \xi^{2})) = (1 - \xi^{2}) \mathbf{r}_{0}(\xi^{1}, 0) + \xi^{1} \mathbf{r}_{0}(\xi^{1}, 1) + (8.23) 
(1 - \xi^{2}) \mathbf{r}_{0}(\xi^{1}, 0) + \xi^{1} \mathbf{r}_{0}(\xi^{1}, 1) 
- (1 - \xi^{1}) (1 - \xi^{2}) \mathbf{r}_{0}(0, 0) + - (1 - \xi^{1}) \xi^{2} \mathbf{r}_{0}(0, 1) 
- \xi^{1} (1 - \xi^{2}) \mathbf{r}_{0}(1, 0) - \xi^{1} \xi^{2} \mathbf{r}_{0}(1, 1)$$

The mapping  $(P_{\xi^1} \oplus P_{\xi^2})$  is the basis for *Trans Finite Interpolation* (TFI) and is used both to generate grids, and also to generate an initial condition for differential grid generation methods.

## 8.3 Differential models for grid generation

## 8.3.1 The Winslow equations

The derivation of the Winslow equations follows directly from the transformation rules for the Christoffel symbol of the second kind:

$$\tilde{\Gamma}^{\alpha}_{\beta\gamma} = \frac{\partial^{2}\xi^{\delta}}{\partial\tilde{\epsilon}^{\beta}\partial\tilde{\epsilon}^{\alpha}}\frac{\partial\tilde{\xi}^{\gamma}}{\partial\xi^{\delta}} + \frac{\partial\xi^{\delta}}{\partial\tilde{\epsilon}^{\alpha}}\frac{\partial\xi^{\sigma}}{\partial\tilde{\epsilon}^{\beta}}\frac{\partial\tilde{\xi}^{\gamma}}{\partial\xi^{\mu}}\Gamma^{\mu}_{\delta\sigma}$$
(8.24)

Take the Cartesian coordinates  $x^{\alpha}$  as the new coordinate system. This means that the Christoffel symbol of the new system vanishes.

$$0 = \frac{\partial^2 \xi^{\delta}}{\partial \tilde{\xi}^{\beta} \partial x^{\alpha}} \frac{\partial x^{\gamma}}{\partial \xi^{\delta}} + \frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\beta}} \frac{\partial x^{\gamma}}{\partial \xi^{\mu}} \Gamma^{\mu}_{\delta \sigma}$$
(8.25)

Next multiply (8.25) with  $\partial \xi^{\nu}/\partial x^{y}$  and contract over  $\gamma$ :

$$0 = \frac{\partial^{2} \xi^{\delta}}{\partial x^{\beta} \partial x^{\alpha}} \frac{\partial x^{\gamma}}{\partial \xi^{\delta}} \frac{\partial \xi^{\nu}}{\partial x^{\gamma}} + \frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\beta}} \frac{\partial x^{\gamma}}{\partial \xi^{\mu}} \frac{\partial \xi^{\nu}}{\partial x^{\gamma}} \Gamma^{\mu}_{\delta\sigma} =$$

$$\frac{\partial^{2} \xi^{\delta}}{\partial x^{\beta} \partial x^{\alpha}} \frac{\partial \xi^{\nu}}{\partial \xi^{\delta}} + \frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\beta}} \frac{\partial \xi^{\nu}}{\partial \xi^{\mu}} \Gamma^{\mu}_{\delta\sigma} =$$

$$\frac{\partial^{2} \xi^{\nu}}{\partial x^{\beta} \partial x^{\alpha}} + \frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\beta}} \Gamma^{\nu}_{\delta\sigma}$$

$$(8.26)$$

Or rearranging:

$$\frac{\partial^2 \xi^{\nu}}{\partial x^{\beta} \partial x^{\alpha}} = -\frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\beta}} \Gamma^{\nu}_{\delta \sigma} \tag{8.27}$$

Contract over  $\alpha$  (multiple subscripts, allowed because the coordinate system is Cartesian!).

$$\frac{\partial^{2} \xi^{\nu}}{\partial x^{\alpha} \partial x^{\alpha}} = \nabla^{2} \xi^{\nu} = -\frac{\partial \xi^{\delta}}{\partial x^{\alpha}} \frac{\partial \xi^{\sigma}}{\partial x^{\alpha}} \Gamma^{\nu}_{\delta \sigma} = -\mathbf{a}^{(\sigma)} \cdot \mathbf{a}^{(\delta)} \Gamma^{\nu}_{\delta \sigma} = -g^{\sigma \delta} \Gamma^{\nu}_{\delta \sigma} = \\
-g^{\sigma \delta} \frac{\partial a^{\alpha}_{(\delta)}}{\partial \xi^{\sigma}} a^{(\nu)}_{\alpha} = \frac{\partial^{2} x^{\alpha}}{\partial \xi^{\delta} \partial \xi^{\sigma}} \frac{\partial \xi^{\nu}}{\partial x^{\alpha}} \tag{8.28}$$

Multiply () by  $\partial x^{\alpha}/\partial x^{\nu}$ . We end up with:

$$\nabla^2 \xi^{\nu} \frac{\partial x^{\alpha}}{\partial \xi^{\nu}} = -g^{\sigma \delta} \frac{\partial^2 x^{\alpha}}{\partial \xi^{\sigma} \partial \xi^{\delta}}$$
 (8.29)

For the curvilinear coordinates the following should hold:

$$\nabla^2 \xi^{\nu} = 0, \quad \nu = 1, 2, 2 \tag{8.30}$$

or using (8.29):

$$g^{\sigma\delta} \frac{\partial^2 x^{\alpha}}{\partial \xi^{\sigma} \partial \xi^{\delta}} = 0, \quad \alpha = 1, 2,$$
 (8.31)

Transforming () to () changed the set of uncoupled linear equations to a set of nonlinear, coupled equations. For solution of () it is advantageous to express the contravariant metric tensor in the covariant metric tensor, as the latter can be calculated directly from the derivatives of the curvilinear coordinates. Using the dummy covariant base vector  $\mathbf{a}_{(3)} = (0,0,1)$ , the contravariant metric tensor can be expressed as:

$$\begin{pmatrix} g^{11} & g^{12} & g^{13} \\ g^{21} & g^{22} & g^{23} \\ g^{31} & g^{32} & g^{33} \end{pmatrix} = \frac{1}{g} \begin{pmatrix} g_{22} & -g_{12} & 0 \\ g_{12} & g_{11} & 0 \\ 0 & 0 & g \end{pmatrix}$$
(8.32)

Using these expressions for the contravariant metric tensor, the *Winslow equations* are given by:

$$g_{22} \frac{\partial^2 x^1}{\partial \xi^1 \partial \xi^1} - 2g_{12} \frac{\partial^2 x^1}{\partial \xi^1 \partial \xi^2} + g_{11} \frac{\partial^2 x^1}{\partial \xi^2 \partial \xi^2} = 0$$

$$g_{22} \frac{\partial^2 x^2}{\partial \xi^1 \partial \xi^1} - 2g_{12} \frac{\partial^2 x^2}{\partial \xi^1 \partial \xi^2} + g_{11} \frac{\partial^2 x^2}{\partial \xi^2 \partial \xi^2} = 0$$

$$(8.33)$$

Both equations for  $x^1$  and  $x^2$  are identical, but of course the solutions are different because of the boundary conditions. The nonlinear system of equations (8.33) can be solved iteratively, by lagging the metric coefficients in each iteration. The coefficients  $g_{ij}$  can be computed from the computed grid functions  $\mathbf{r}_0(\xi^1,\xi^2)$  using second order central differencing. As an example consider the computation of  $g_{12}$  in grid point  $(\xi_i^1,\xi_j^2)$ .

$$(g_{12})_{ij} = \left[\frac{\partial x^1}{\partial \xi^1} \frac{\partial x^1}{\partial \xi^2}\right]_{ij} + \left[\frac{\partial x^2}{\partial \xi^1} \frac{\partial x^2}{\partial \xi^2}\right]_{ij} = \tag{8.34}$$

$$\frac{\left(x_{i+1j}^{1}-x_{i-1j}^{1}\right)\left(x_{ij+1}^{1}-x_{ij-1}^{1}\right)}{4\Delta\xi^{1}\Delta\xi^{2}}+\frac{\left(x_{i+1j}^{2}-x_{i-1j}^{2}\right)\left(x_{ij+1}^{2}-x_{ij-1}^{2}\right)}{4\Delta\xi^{1}\Delta\xi^{2}}+\mathcal{O}((\Delta\xi)^{2})$$

## 8.3.2 The Thompson-Thames-Mastin (TTM) equations

With the Winslow equations no control over grid density is possible beyond changing the parameterization of the boundary curves. To cluster grid lines in the vicinity of the boundary to accurately capture boundary layers, e.g. in high Reynolds number flow we introduce source terms. Additionally, source terms can be defined to control the direction of the grid lines in the vicinity of the boundary, either to improve the accuracy of the discretisation or to enable  $C^1$  continuity of the grid lines across block-interfaces in a multi-block grids. The inhomogeneous form of the Winslow equations is commonly referred to as the Thompson-Thames-Mastin (TTM) equations and is given by

$$\nabla^2 \xi^1 = P(\xi^1, \xi^2) \tag{8.35}$$

$$\nabla^1 \xi^2 = Q(\xi^1, \xi^2) \tag{8.36}$$

Using (8.35) the transformed equations:

$$\nabla^2 \xi^{\nu} \boldsymbol{a}_{(\nu)} = -g^{\delta \sigma} \frac{\partial^2 \xi^{\alpha}}{\partial \xi^{\sigma} \partial \xi^{\delta}}, \tag{8.37}$$

now become

$$Pa_{(1)}^{\alpha} + Qa_{(2)}^{\alpha} = -g^{\delta\sigma} \frac{\partial^2 \xi^{\alpha}}{\partial \xi^{\sigma} \partial \xi^{\delta}}, \tag{8.38}$$

where the dependence of  $(\xi^1, \xi^2)$  of P and Q is not explicitly denoted. Transforming to the formulation using the co-variant metric tensor as opposed to the contravariant metric tensor leads to the standard form of the TTM equations.

$$g_{22} \frac{\partial^2 x^{\alpha}}{\partial \xi^1 \partial \xi^1} - 2g_{12} \frac{\partial^2 x^{\alpha}}{\partial \xi^1 \partial \xi^2} + g_{11} \frac{\partial^2 x^{\alpha}}{\partial \xi^2 \partial \xi^2} = -g \left( P \frac{\partial x^{\alpha}}{\partial \xi^1} + Q \frac{\partial x^{\alpha}}{\partial \xi^2} \right)$$
(8.39)

Two types of source functions are of interest in this context:

- Source functions to control grid line clustering in the vicinity of grid points or grid lines.
- Source functions to control the direction of the grid lines in the vicinity of the boundary: an artificial way to impose a Neumann condition on the mapping.

## 8.3.3 Source function for control of grid density

The common form of the two source functions P and Q in the TTM equations to achieve grid clustering of grid lines in the vicinity of the grid lines  $\xi^{\alpha} = \xi_n^{\alpha}$  and in the vicinity of the points  $(\xi_i^1, \xi_i^2)$  is given by:

$$P(\xi^{1}, \xi^{2}) =$$

$$\sum_{n=1}^{N} \frac{\xi^{1} - \xi_{n}^{1}}{|\xi^{1} - \xi_{n}^{1}|} e^{-c_{n}|\xi^{1} - \xi_{n}^{1}|} - \sum_{i=1}^{L} b_{i} \frac{\xi^{1} - \xi_{i}^{1}}{|\xi^{1} - \xi_{i}^{1}|} e^{-d_{i}\left[(\xi^{1} - \xi_{i}^{1})^{2} + (\xi^{2} - \xi_{i}^{2})^{2}\right]}$$

$$Q(\xi^{1}, \xi^{2}) =$$

$$\sum_{n=1}^{N} \frac{\xi^{2} - \xi_{n}^{2}}{|\xi^{2} - \xi_{n}^{2}|} e^{-c_{n}|\xi^{2} - \xi_{n}^{2}|} - \sum_{i=1}^{L} b_{i} \frac{\xi^{2} - \xi_{i}^{2}}{|\xi^{2} - \xi_{i}^{2}|} e^{-d_{i}\left[(\xi^{1} - \xi_{i}^{1})^{2} + (\xi^{2} - \xi_{i}^{2})^{2}\right]}$$

$$(8.41)$$

For positive(negative) values of  $a_n/b_i$  grid lines are attracted(repelled) from the grid lines  $\xi^{\alpha} = \xi_n^{\alpha}$  and the grid points  $(\xi_i^1, \xi_i^2)$ . The control of the grid lines is done in the *logical* domain, not in the physical domain which would be much more desirable. Of course we would rather be able to cluster lines around a certain point  $(x_c^1, x_c^2)$  or in the vicinity of a line  $x^{\alpha} = x_c$  when we expect a strong gradient in the solution on that location. The logical space grid control makes designing a grid a little less transparent than wanted.

## 8.3.4 Source function for control of direction of the grid lines

In many instances we would like to be able to control the direction of the grid lines, most likely make them orthogonal to the boundary. Clearly this is not directly possible within the framework of either the Winslow or TTM equations. Both are elliptic second order partial differential equations and allow the prescription of a single boundary condition on the boundary of the domain. Of course we need to impose a Dirichlet boundary condition on all four sides of the domain to have the grid match up with the four parameterized curves that comprise the boundary of the domain. To be able to impose an additional Neumann boundary condition to control the direction of the grid lines we would have to raise the order of our partial differential equation to four:e.g. the bi-harmonic equation. The amount of work and computational resource required to solve the bi-harmonic equation and the added complexity of handling the large discretisation stencil in the vicinity of the boundary make that this approach is seldom put in practice. If we stick to our Winslow or TTM equations the only way to impose a Neumann condition on the boundary is indirectly, by imposing source terms in the vicinity of the boundary. As an example we will show to impose a Neumann boundary condition on the lower boundary of the domain ( $\xi^2 = 0$ ). We impose:

$$\frac{\partial \mathbf{x}}{\partial \xi^2} = \mathbf{a}_{(2)} = \frac{1}{\sqrt{g_{11}}} \left[ \sigma^1(\xi^1) \frac{\partial \mathbf{x}}{\partial \xi^1} + \sigma^2(\xi^1) \left( \frac{\partial \mathbf{x}}{\partial \xi^1} \right)^{\perp} \right], \tag{8.42}$$

where we introduced the vector  $\left(\frac{\partial \boldsymbol{x}}{\partial \xi^1}\right)^{\perp}$ , defined as:

$$\left(\frac{\partial \boldsymbol{x}}{\partial \xi^{1}}\right)^{\perp} \equiv = \hat{\boldsymbol{n}} \times \frac{\partial \boldsymbol{x}}{\partial \xi^{1}} = \sqrt{g}\boldsymbol{a}^{(1)}$$
(8.43)

to have a nice symmetric formulation of the boundary condition. We are not completely free in our definition of the functions  $\sigma^1(\xi^1)$  and  $\sigma^2(\xi^1)$ , because the direction of the grid lines should match the direction of the boundary curves for the two vertices (0,0) and (1,0), as indicated in Figure \*\*. This requirement leads to the following two consistency conditions:

$$\frac{1}{\sqrt{g_{11}}} \left[ \sigma^1(0) \left. \frac{\partial \mathbf{x}}{\partial \xi^1} \right|_{(0,0)} + \sigma^2(0) \left( \left. \frac{\partial \mathbf{x}}{\partial \xi^1} \right)^{\perp} \right|_{(0,0)} \right] = \left. \frac{\partial \mathbf{x}}{\partial \xi^2} \right|_{(0,0)}$$
(8.44)

$$\frac{1}{\sqrt{g_{11}}} \left[ \sigma^{1}(1) \left. \frac{\partial \boldsymbol{x}}{\partial \xi^{1}} \right|_{(1,0)} + \sigma^{2}(1) \left( \left. \frac{\partial \boldsymbol{x}}{\partial \xi^{1}} \right)^{\perp} \right|_{(1,0)} \right] = \left. \frac{\partial \boldsymbol{x}}{\partial \xi^{2}} \right|_{(1,0)}$$
(8.45)

The main question is of course how to derive the appropriate source terms in such a way that the Neumann boundary condition is enforced. Recall that the TTM equations are given by:

$$g_2 2 \frac{\partial^2 x^{\alpha}}{\partial \xi^1 \partial \xi^1} - 2g_{12} + g_{11} \frac{\partial^2 \xi^{\alpha}}{\partial \xi^2 \partial \xi^2} = -g \left( P \frac{\partial x^{\alpha}}{\partial \xi^1} + Q \frac{\partial x^{\alpha}}{\partial \xi^2} \right)$$
(8.46)

The general form of the source terms P and Q for this application is chosen as:

$$P(\xi^1, \xi^2) = \varphi_0(\xi^1)e^{-\lambda \xi^2} \tag{8.47}$$

$$Q(\xi^1, \xi^2) = \psi_0(\xi^1)e^{-\lambda \xi^2} \tag{8.48}$$

(8.49)

The factor  $e^{-\lambda \xi^2}$  'blends' the source term with the interior of the domain. Imposing the source terms will adversely affect the smoothness of the grid in the interior of the domain, so the sources are only applied where needed.

We can abbreviate (8.46) in the following way:

$$Qx^{\alpha} = -g \left[ \varphi_0(\xi^1) a_{(1)}^{\alpha} + \psi_0(\xi^1) a_{(2)}^{\alpha} \right] e^{-\lambda \xi^2}$$
 (8.50)

or in vector form:

$$Qx = -g \left[ \varphi_0(\xi^1) a_{(1)} + \psi_0(\xi^1) a_{(2)} \right] e^{-\lambda \xi^2}$$
(8.51)

To derive the explicit expression for the two source terms  $\varphi_0(\xi^1)$  and  $\psi_0(\xi^1)$  we require that (8.51) holds on the boundary  $\xi^2 = 0$ =

$$Qx = -g \left[ \varphi_0(\xi^1) a_{(1)} + \psi_0(\xi^1) a_{(2)} \right]$$
 (8.52)

We can express the operator Q working on the mapping x in the following way:

$$Qx = g_{22} \frac{\partial^{2} x}{\partial \xi^{1} \partial \xi^{1}} - 2g_{12} \frac{\partial^{2} x}{\partial \xi^{1} \partial \xi^{2}} + g_{11} \frac{\partial^{2} x}{\partial \xi^{2} \partial \xi^{2}}$$

$$= g_{22} \frac{\partial \mathbf{a}_{(1)}}{\partial \xi^{1}} - 2g_{12} \frac{\partial \mathbf{a}_{(1)}}{\partial \xi^{2}} + g_{11} \frac{\partial \mathbf{a}_{(2)}}{\partial \xi^{2}}$$

$$= (g_{22} \Gamma_{11}^{k} - 2g_{12} \Gamma_{12}^{k} - g_{11} \Gamma_{22}^{k}) \mathbf{a}_{(k)}$$
(8.53)

To compute the functions  $\varphi_0(\xi^1)$  and  $\psi_0(\xi^1)$  we take the inner product of (8.53) with  $a^{(1)}$  and  $a^{(2)}$ , respectively.

$$\mathbf{a}^{(1)} \cdot Q\mathbf{x} = -g \left[ \varphi_0(\xi^1) \mathbf{a}_{(1)} + \psi_0(\xi^1) \mathbf{a}_{(2)} \right] \cdot \mathbf{a}^{(1)} = -g \varphi_0(\xi^1)$$
 (8.54)

$$\mathbf{a}^{(2)} \cdot Q\mathbf{x} = -g \left[ \varphi_0(\xi^1) \mathbf{a}_{(1)} + \psi_0(\xi^1) \mathbf{a}_{(2)} \right] \cdot \mathbf{a}^{(2)} = -g \psi_0(\xi^1)$$
(8.55)

Denote that:

$$\Gamma_{ij}^{k} \boldsymbol{a}^{(l)} \cdot \boldsymbol{a}_{(k)} = \Gamma_{ij}^{k} \delta_{k}^{l} = \Gamma_{ij}^{l}$$
(8.56)

And we end up with the following expressions for the two source terms:

$$\frac{1}{g} \left( g_{22} \Gamma_{11}^1 - 2g_{12} \Gamma_{12}^1 + g_{11} \Gamma_{22}^1 \right) = \varphi_0(\xi^1)$$
 (8.57)

$$\frac{1}{g} \left( g_{22} \Gamma_{11}^2 - 2g_{12} \Gamma_{12}^2 + g_{11} \Gamma_{22}^2 \right) = \psi_0(\xi^1)$$
 (8.58)

To impose these two source terms we need the following geometrical quantities on the boundary

$$\begin{pmatrix}
g_{11} & g_{12} & g_{22} \\
\Gamma_{11}^{1} & \Gamma_{12}^{1} & \Gamma_{22}^{1} \\
\Gamma_{11}^{2} & \Gamma_{12}^{2} & \Gamma_{22}^{2} \\
\sqrt{g}
\end{pmatrix}$$
(8.59)

Nearly all of these quantities can be computed from the parameterization of the boundary curve combined with the prescribed Neumann boundary condition, for example:

$$g_{11} = \mathbf{a}_{(1)} \cdot \mathbf{a}_{(1)} \tag{8.60}$$

$$g_{12} = \mathbf{a}_{(1)} \cdot \mathbf{a}_{(2)} = \tag{8.61}$$

$$\boldsymbol{a}_{(1)} \cdot \frac{1}{\sqrt{g}} \left( \sigma_1(\xi^1) \boldsymbol{x}_{\xi^1} + \sigma_2(\xi^1) (\boldsymbol{x}_{\xi^1})^\perp \right) = \sigma_1(\xi^1) \frac{1}{\sqrt{g}}$$

$$g_{22} = \left[ \frac{1}{\sqrt{g}} \left( \sigma_1(\xi^1) \boldsymbol{x}_{\xi^1} + \sigma_2(\xi^1) (\boldsymbol{x}_{\xi^1})^{\perp} \right) \right]^2 = (\sigma_1(\xi^1))^2 + (\sigma_2(\xi^1))^2$$
 (8.62)

$$\sqrt{g} = |\boldsymbol{a}_{(1)} \times \boldsymbol{a}_{(1)}| = \tag{8.63}$$

$$egin{aligned} \sqrt{g} &= |oldsymbol{a}_{(1)} imes oldsymbol{a}_{(1)}| = \ |oldsymbol{x}_{\xi^1} imes rac{1}{\sqrt{g}} \left(\sigma_1(\xi^1) oldsymbol{x}_{\xi^1} + \sigma_2(\xi^1) (oldsymbol{x}_{\xi^1})^\perp
ight) = \sqrt{g_{11}} \sigma_2(\xi^1) \end{aligned}$$

One difficulty remains in the geometrical quantities that depend on second order mixed or pure derivatives in the  $\xi^2$  direction. Those can not be computed either from the parameterization of the curve or the Neumann boundary condition. The only option is to use a one-sided finite difference approximation into the domain and lag the computation of the source term behind in the iterative process. The major difference between the iterative process for the Winslow equations and the TTM equations with grid clustering sources is the fact that we are now lagging coefficients dependent on second order derivatives, as opposed to the first order derivatives that comprise the coefficients of the covariant metric tensor. This difference causes a significant difference in convergence rate between the two iterations: prescribing orthogonality comes at a price.

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## $\mathbf{Index}$

	GG 107 100
absolute stability, 70	CG, 125, 128
accuracy	chaos, 11
- uniform, 60	characteristics, 45, 50, 52
Adams-Bashforth scheme, 85, 95	Cholesky decomposition, 120
Adams-Bashforth- $\omega$ -scheme, 99	- incomplete, 120, 121
Adams-Bashforth-Crank-Nicolson	cholinc, 120, 121
scheme, 95, 98	cholinc, 127
Adams-Bashforth- $\omega$ scheme, 85	compatibility condition, 88, 97
aircraft, 11	condition number, 108, 126
amplification factor, 74	conjugate gradients method, 125
artificial viscosity, 30	conservation
	- of mass, 5, 6
back-substitution, 110, 118	of momentum, 7, 8
backward-facing step problem, 99	conservation form, 18, 19, 43, 86
bandmatrix, 104	conservation law, 5
bandwidth, 109, 115	conservative, 23, 55
- lower, 109, 110	consistency, 68–70
- upper, 109, 110	constitutive relation, 8
barrier function, 38, 40	continuity equation, 6, 9
basic iterative method, 110, 116, 128	continuum hypothesis, 5
benchmark, 114, 115	control volume, 22
BIM, 116–118, 123, 124, 129, 132	convection-diffusion equation, I, 13, 14,
Black-Scholes equation, 13	17, 53
body force, 7, 9	- dimensionless form, 14
boundary condition, 19, 20, 24, 49, 85, 91	convection-diffusion-reaction equation,
- Dirichlet, 19, 20, 24	13
- Neumann, 19, 20, 25	convergence, 68–70, 110, 115–117, 119,
- wrong, 20	124, 125, 128, 130, 132
boundary layer, 31, 44	- of discretization, 68
- equation, 46, 47, 51, 52	- rate of —, 121
- ordinary, 52	coordinates
- parabolic, 52, 54	- Cartesian, 2
- thickness, 31	- right-handed, 2
boundary value problem, 19	Courant number, 77
Burgers equation, 19, 72	Courant-Friedrichs-Lewy number, 77
<b>3</b> 1 , ,	Crank-Nicolson scheme, 78
Cartesian grid, 99	cruise condition, 11
Cartesian tensor notation, 2	curl, 4
cavitation, 7	0411, 1
cd1, 27, 28, 42	d'Alembert's paradox, 46
cd2, 58, 59	determinism, 12
cdns, 81	deviatoric stress tensor, 86
cell, 22	diagonally dominant, 109, 110
cell-centered, 23, 28, 37, 41, 89	differential-algebraic system, 93, 94, 97
central discretization, 24	diffusion coefficient, 13
central scheme, 25	diffusion number, 65
CFL number, 77	dimensionless
,	amonoiomos

- convection-diffusion equation, 14 - equations, 9 - Navier-Stokes equations, 10, 86 - parameter, 10, 11 - variables, 10 direct method, 110, 112, 116 direct solver, 104 Dirichlet, 52, 53 Dirichlet condition, 19 distinguished limit, 49, 51, 52 distributive iteration, 96, 110, 128, 130, 132 divergence theorem, 2 driven cavity, 113  efficiency, 66, 85, 103, 110, 112, 117, 119, 127, 128 eigenvalue, 108, 123 eigenvector, 108, 117, 123 elliptic, 45 ERCOFTAC, 1 errata, II Euler scheme - backward, 77 - forward, 65, 72, 76, 77, 94 explicit, 77, 79 explicit scheme, 71  Fick's law, 13 fill-in, 111, 119 finite difference, I finite element, I finite volume, I, 22, 54, 90 finite volume method, 17, 22, 44 fish, 11 floor, 58 flops, 105, 110, 111, 123 flux, 23, 24 flux vector, 13 Fourier - analysis, 71 - law, 13 - mode, 73 - series, 72 - stability analysis, 72, 98 fractional step method, 96	- first order, 120 ICCG, 126, 127 ill-posed, 17, 20 ILU, 110, 118 ILU decomposition, 119, 121 - first order, 119, 121 IMEX method, 104 IMEX scheme, 95 implicit, 77 incomplete Cholesky decomposition, 120, 121 - first order —, 120 incomplete LU method, 118 incompressiblity, 7 incompressible, 6 induced norm, 108 inertia, 8, 11 inertia terms, 9 inflow boundary, 50 inflow conditions, 87, 92 initial condition, 86 inner equation, 47, 49 inner product, 2 inner solution, 47, 48 Internet, 1, 107 irreducible, 109 irrotational, 4 iteration matrix, 116 iterative method, 110, 112, 116, 121 - backward Gauss-Seidel, 118 - basic, 116, 128 - distributive, 128, 130, 132 - forward Gauss-Seidel, 118 - Gauss-Seidel, 118, 124, 132 - ILU, 118 - Jacobi, 118, 122, 124, 132 - SIMPLEC, 132 - SIMPLER, 132 - stationary, 116, 130  Jacobi method, 110, 118, 122, 124, 132 Jacobian, 94  K-matrix, 109, 110, 117, 118, 128, 129 kron, 121
Gauss-Seidel method, 110, 118, 124, 132  – backward, 118, 124  – forward, 118, 124  Gaussian elimination, 110, 118  heat equation, 64  heq, 66  hyperbolic, 45  IC, 120	Lagrange multiplier, 94 laminar, I, 1, 11 Landau's order symbol, 35 Laplace, 12, 127 Laplace operator, 2, 9 Lax's equivalence theorem, 70 length scale, 81 Leonardo da Vinci, 11 lexicographic order, 58, 119, 121 linear interpolation, 24

linearization, 94, 112 - extrapolated Picard, 95, 99 - Newton, 85, 94, 99 - Picard, 85, 95, 99 local grid refinement, 33 local mesh refinement, 60 lower triangular, 109 1u, 111 LU-decomposition, 104 luinc, 121	- induced —, 108 - matrix —, 108 - p- —, 108 - vector —, 108 ns1, 85, 99 ns2, 85, 99 ns3, 112 numerical diffusion, 30 numerical efficiency, 103 numerical flux, 24
M-matrix, 109, 117–119, 124, 128 matching principle, 47, 49, 51 material  – particle, 5, 6, 13  – property, 5, 13  – volume, 5–7 mathematical finance, 13 MATLAB, 18, 27, 41, 56, 58, 59, 66, 99,	ω-scheme, 77, 78, 80, 85, 99 one-step scheme, 68 outer equation, 47, 50, 51 outer solution, 47, 48, 50 outflow, 85 outflow boundary, 53, 54 outflow condition, 17, 52, 88, 92 overrelaxation, 116
103, 125 MATLAB code, 112 MATLAB software, II matrix - diagonally dominant, 109, 110 - irreducible, 109 - K- —, 109, 110, 117, 118, 128, 129 - M- —, 109, 117-119, 124, 128 - positive definite, 109, 110 - sparse, 109, 110 - transpose, 108 - triangular, 109 matrix norm, 108 maximum principle, 17, 21, 29, 43, 57, 63, 117 - discrete — 17, 44	p-norm, 108 Péclet  - mesh — number, 30, 57, 61, 105  - number, 14, 28, 33, 54  - uniform, 33  - uniform accuracy, 54  - uniform computing cost, 54  parabolic, 51  paradox of d'Alembert, 46  Parseval's theorem, 74  Pentium, 67, 103  physical units, 9  Picard, 85, 95, 99  - iteration, 112, 128  - linearization, 112
- discrete —, 17, 44 mesh Péclet number, 30, 57, 105 mesh Reynolds number, 105, 129 mesh size, 23 momentum equation, 8 monotone, 17, 21 multigrid, 107, 112, 125, 132 multiphase flow, 6 my_cholinc, 120, 127  Navier-Stokes equations, I, 9, 85	pivoting, 110 po, 121, 126, 127 Poisson equation, 97, 120, 122, 132 positive definite, 109, 110 positive scheme, 29 positive type, 29, 57, 117 postconditioning, 130 potential, 4 — flow, 3 preconditioning, 125
- dimensionless, 10, 86 - incompressible, 9 - nonstationary, 107 - stationary, 107 Neumann, 52, 53 Neumann condition, 19 Newton, 1, 7, 85, 94, 99 Newtonian - fluid, 8 - mechanics, 12 - non- —, 8 no-slip, 87, 92 norm - l <sub>2</sub> —, 74	predictability of dynamical systems, 12 pressure Poisson equation, 97 pressure-correction method, 85, 96, 98, 129 projection method, 96  random, 12 rate of convergence, 121 rate of strain, 8 rate of strain tensor, 8 regular perturbation, 46 regular splitting, 110, 117, 124, 128, 129 relaxation parameter, 116 reordering, 105, 110, 111 residual, 117

Reynolds	symmetry, 27
- mesh — number, 105, 129	symmetry, 21
- number, 1, 10–12, 33, 54, 61, 86	Taylor's formula, 34, 35
- transport theorem, 5	tensor
rotation, 4	- rate of strain —, 8
rounding error, 34	- stress —, 8
	termination criterion, 117
scheme	time scale, 81
- central, 25	total derivative, 5
- finite volume, 26	transport theorem, 5, 6
- numerical, 26	transpose, 108
- upwind, 25	truncation error, 34
sea-level, 11 ship, 11	<ul> <li>global, 18, 34, 68, 114, 116</li> <li>local, 17, 34, 35, 68, 69, 114</li> </ul>
SIMPLE, 110, 129, 130, 132	turbulence, 11
termination criterion, 132	turbulent, I
SIMPLEC, 132	two-step method, 95
SIMPLER, 132	r,
singular perturbation, 34, 46	underrelaxation, 116, 129
singular perturbation theory, 44, 46, 47	uniformly valid, 50
smooth grid, 35, 36	upper triangular, 109
software, 107	upwind discretization, 24
software libraries, 108	upwind scheme, 25, 72, 76, 115, 129
solenoidal, 3, 4, 94	
sparse, 27, 104, 109, 110	vector
spdiags, 27, 121	- analysis, 1
spectral method, I	- norm, 108
spectral radius, 108, 123, 124 speed of sound, 7	- notation, 2 - space, 108
splitting, 129, 130	vectorization, 66
- convergent, 117	vertex-centered, 23, 27, 40, 41, 89
- regular, 117, 118	virtual value, 92, 93
spy, 111, 121	viscosity
stability, 68, 70, 78, 81, 85, 98	- coefficient, 86
- absolute, 70, 75, 76	- dynamic, 8
- analysis, 71, 78	- kinematic, 8, 11
- Fourier analysis, 71, 72, 98	– numerical, 33
- local, 71	von Neumann condition, 75
- zero- —, 70, 75, 76	11 1 1 4 100
staggered, 88, 89, 91, 94	wall-clock time, 103
staggered grid, 85 standard atmosphere, 11	wavenumber, 73
stationary iterative method, 116	website, II, 1 well-posed, 17, 20, 51
stencil, 26, 44, 57, 91, 119, 121	wiggles, 21, 29, 60, 61, 64, 88, 117
Stokes	windtunnel, 11
- equations, 12	wing cord, 11
- paradox, 12	wp_iccg, 127
stratified flow, 8	• •
streamfunction, 3, 114, 115	yacht, 11
streamline, 3, 4, 114	
stress tensor, 8	zero-stability, 70
- deviatoric —, 86	
stretched coordinate, 47	
subcharacteristics, 45	
subscript notation, 2 summation convention, 2	
surface force, 7	
swapping, 114	
symmetric positive definite, 125	
,	