

## Fluid Mechanics

Density:  $\rho = \frac{M}{V}$ ; pressure:  $p = \frac{\Delta F \perp}{\Delta A}$ ; force of buoyancy:  $F_B = \rho_{fluid} V_{object} g$ ; mass conservation means:  
 $\rho A v = Const.$

For incompressible fluids:  $\rho = Const.$  Bernoulli's equation for any fluid:

$$\frac{p}{\rho} + \frac{1}{2} v^2 + gh = Const.$$

Power flow (with flow  $Q = Av$ ):

$$pQ + \frac{1}{2} \rho v^2 Q + \rho g h Q = Const.$$

Distribution of speeds in a cylindrical pipe due to viscosity:

$$v = \frac{P_1 - P_2}{4\eta l} (R^2 - r^2)$$

where  $R$  is the radius of the pipe and  $r$  is the radial distance from the center;  $P_1$  is the pressure at one end;  $l$  is the length of the pipe and  $\eta$  is the viscosity of the fluid. For laminar flow, the flow rate  $Q$  in  $m^3/s$  is given by:

$$Q = \frac{\pi R^4}{8\eta l} (P_1 - P_2)$$

The force needed to stretch the surface area is

$$F = 2\gamma l$$

where, the length is  $l$  and  $\gamma$  is the surface tension. Pressure across a drop of liquid ( $\delta P$ ): consider a spherical water drop of radius  $R$

$$\delta P = \frac{2\gamma}{R}$$

Capillary action: height of liquid inside a tube of radius  $r$  and contact angle  $\theta$

$$h = \frac{2\gamma \cos \theta}{\rho g r}$$

## Thermodynamics

Linear expansion:  $\Delta L = \alpha L_0 \Delta T$ ; volume expansion:  $\Delta V = \beta V_0 \Delta T$

Specific Heat:  $\Delta Q = mc\Delta T$ ; in term of moles ( $n$ ):  $\Delta Q = nC\Delta T$ ; the mass of a mole  $M$  then  $m = nM$

Heat transfer in change of phase:  $\Delta Q = mL$

Heat transfer. Conduction:  $H = \frac{\Delta Q}{\Delta t} = kA \frac{\Delta T}{L}$ ; the R factor  $R = \frac{L}{k}$ .

Radiation:  $H = Ae\sigma T^4$  where  $\sigma = 5.67 \times 10^{-8}$ ; Wien's law:  $f_{max} = 5.879 \times 10^{10} T$

Dependence of entropy on energy:  $\frac{1}{T} = \frac{ds}{dE}$ . Alternatively:  $\Delta S = \int \frac{dQ}{T}$

$$T_F = \frac{9}{5}T_C + 32 \quad T = T_C + 273.15$$

## Ideal Gas

$$C_V = \frac{d}{2}k_B$$

$$\frac{C_P}{C_V} = \gamma = 1 + \frac{2}{d}$$

$$v_{rms} = \sqrt{\frac{3k_B T}{m}}$$

$$PV = Nk_B T$$

$$\Delta Q = \Delta W + \Delta E$$

$$E_{th} = \frac{d}{2}Nk_B T$$

$$\Delta W = \int P dV$$

$$S/k_B = N \ln V + \frac{d}{2}N \ln T + C$$

**Isochoric:**

$$\Delta W = 0 \quad \Delta E_{th} = \Delta Q \quad \Delta Q = C_V N \Delta T \quad \Delta E_{th} = N C_V \Delta T$$

$$\Delta S = N C_V \ln \frac{T_f}{T_i} \quad \frac{T_i}{P_i} = \frac{T}{P}$$

**Isentropic:**

$$\Delta S = 0 \quad P_i V_i^\gamma = P V^\gamma \quad \text{or} \quad T_i V_i^{\gamma-1} = T V^{\gamma-1} \quad \Delta Q = 0$$

$$\Delta E_{th} = -\Delta W = N C_V \Delta T \quad \Delta W = \frac{1}{\gamma-1} (P V - P_i V_i)$$

**Isobaric:**

$$\Delta Q = C_P N \Delta T \quad C_P = C_V + k_B \quad \frac{T_i}{V_i} = \frac{T}{V} \quad \Delta S = N C_P \ln \frac{T_f}{T_i} \quad \Delta W = P(V_f - V_i)$$

**Isothermal:**

$$\Delta E_{th} = 0 \quad \Delta Q = \Delta W \quad \Delta W = N k_B T \ln \frac{V_f}{V_i}$$

$$\Delta S = k_B N \ln \frac{V_f}{V_i} \quad P_i V_i = P V$$

**Other processes:**

Specific heat capacity:  $\Delta Q = m c_V \Delta T$  Associate change in entropy:  $\Delta S = m c_V \ln \frac{T_f}{T_i}$  (for warming or cooling)

for a change of phase (at a constant temperature):  $\Delta S = \frac{\Delta Q}{T}$

so, for melting or vaporization:  $\Delta S = \frac{m L}{T}$  and with negative sign for solidification or condensation.

**Heat engines and heat pumps:**

Working in a cycle at the end we have:  $\Delta E_{th} = 0$ ,  $\Delta Q_{Total} = -|\Delta W_{Total}|$

$\Delta S = \Delta S_{device} + \Delta S_{environment} \geq 0$  but the device goes back to original so  $\Delta S_{device} = 0$ ; so  $\Delta S_{environment} \geq 0$

$$e = \frac{|\Delta W|}{|\Delta Q_{in}|} \quad e = 1 - \frac{|\Delta Q_{out}|}{|\Delta Q_{in}|} \quad \text{Carnot cycle: } e_c = 1 - \frac{T_L}{T_H} \quad \text{Brayton cycle: } e = 1 - \left( \frac{P_H}{P_L} \right)^{(1/\gamma)-1}$$

$$\text{Otto cycle } e = 1 - \frac{1}{r^{\gamma-1}}$$

Refrigerators and heat pumps (because the cycle is reverse compared to heat engines):

$$|\Delta Q_{out}| > |\Delta Q_{in}| \quad T_H > T_L \quad |\Delta W| = |\Delta Q_{out}| - |\Delta Q_{in}|$$

$$\begin{aligned} C.O.P_{cooling} &= \frac{|\Delta Q_{in}|}{|\Delta W|} & C.O.P_{heating} &= \frac{|\Delta Q_{out}|}{|\Delta W|} \\ C.O.P_{cooling,max} &= \frac{T_L}{T_H - T_L} & C.O.P_{heating,max} &= \frac{T_H}{T_H - T_L} \end{aligned}$$