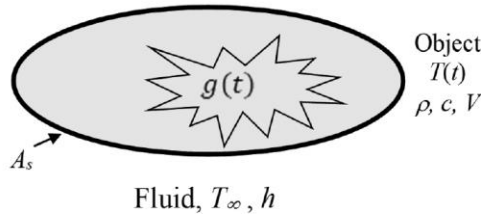


Ordinary Differential Equations: Lumped Thermal Mass

Consider the cooling of a hot object initially at temperature T_0 in cold air at T_∞ with heat transfer coefficient h .



$T(t)$ = temperature (K or $^{\circ}\text{C}$)
 $g(t)$ = heat source per volume ($\text{W}/\text{m}^3 = \text{J}/\text{s}\cdot\text{m}^3$)
 V = volume (m^3)
 A_s = surface area (m^2)
 ρ = mass density (kg/m^3)
 c = specific heat ($\text{J}/\text{kg}\cdot\text{K}$ or $\text{J}/\text{kg}\cdot^{\circ}\text{C}$)
 T_∞ = fluid temperature (K or $^{\circ}\text{C}$)
 h = heat transfer coefficient ($\text{W}/\text{m}^2\cdot\text{K}$ or $\text{W}/\text{m}^2\cdot^{\circ}\text{C}$)

Assume that the lumped thermal capacity model, as described in Vick Ch. 7.2.4, is valid. Applying Conservation of Energy yields the differential equation:

$$\rho c V \frac{dT}{dt} = -h A_s (T - T_\infty) + gV$$

- Draw the cause-effect diagram for this physical problem.
- Determine the steady-state solution (aka, find the fixed point). Is it stable or unstable? Sketch the phase portrait and anticipated solution.
- Write a function to numerically solve for temperature as a function of time. The system parameters and forcing functions should be inputs, and the time and temperature vectors should be outputs.
- Explore the effect of the heat transfer coefficient, h . Plot temperature versus time for $h = 0 \frac{\text{W}}{\text{m}^2 \cdot ^{\circ}\text{C}}$ (no heat transfer), $10 \frac{\text{W}}{\text{m}^2 \cdot ^{\circ}\text{C}}$ (still day), $25 \frac{\text{W}}{\text{m}^2 \cdot ^{\circ}\text{C}}$ (typical day), and $75 \frac{\text{W}}{\text{m}^2 \cdot ^{\circ}\text{C}}$ (hurricane). Put all curves on a single graph. Carry out the integration long enough that the temperature begins to level out at steady state and use a step size of 1 s. The other parameters are $V = 10^{-6} \text{ m}^3$, $A_s = 10^{-4} \text{ m}^2$, $\rho = 1000 \frac{\text{kg}}{\text{m}^3}$, $c = 500 \frac{\text{J}}{\text{kg}}$, $T_\infty = 25^{\circ}\text{C}$, $T_0 = 400^{\circ}\text{C}$, and $g(t) = 0 \frac{\text{W}}{\text{m}^3}$.
- Explore the effect of a *constant* heat source strength, $g(t)$. Plot the solution for $g(t) = 0, 10^5, 2 * 10^5$, and $3 * 10^5 \frac{\text{W}}{\text{m}^3}$. Put all curves on a single graph. Use the parameter values from part (d) with $h = 25 \frac{\text{W}}{\text{m}^2 \cdot ^{\circ}\text{C}}$.

f) Explore the effect of a *pulsed* heat source, defined by:

$$g(t) = g_c \frac{1}{\Delta t} (H(t) - H(t - \Delta t))$$

g_c = heat source strength $\left(\frac{J}{m^3}\right)$, Δt = pulse time (s), and $H(t)$ = unit step/Heaviside function. Plot the temperature of the thermal mass for:

$$g_c = 4 * 10^7 \frac{J}{m^3}$$

$$\Delta t = 400, 200, 100 \text{ s}$$

$$T_0 = T_\infty = 25 \text{ }^\circ\text{C}$$

$$h = 25 \frac{W}{m^2 \text{ }^\circ\text{C}}$$

The remaining parameters are the same as in part (d). For this part, employ a relative and absolute tolerance of 10^{-9} . Create a subplot containing the heat source $g(t)$ on the upper subplot and $T(t)$ on the lower subplot. Physically interpret your results.

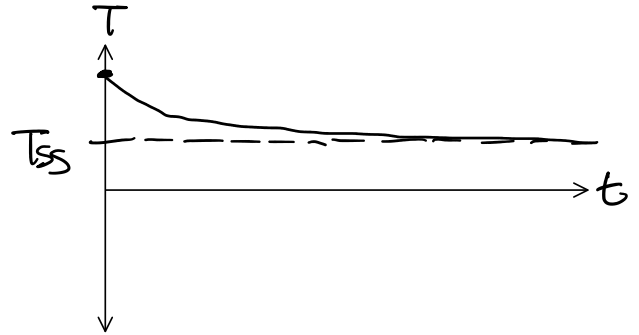
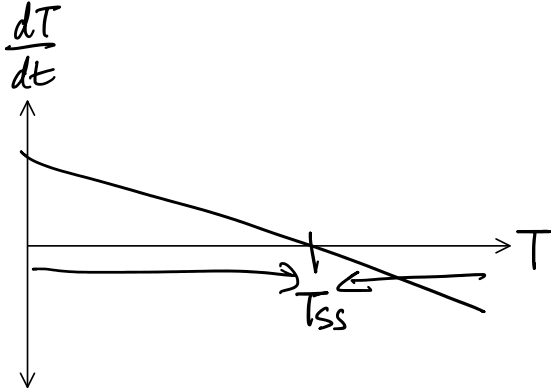
Lumped Thermal Mass Phase Portrait/Anticipated Solution

ODE and Initial Condition	Fixed Points (=Steady-State Values)	Stability
$\rho c V \frac{dT}{dt} = -h A_s (T - T_\infty) + g V$ $T_0 = 400\text{ }^\circ\text{C}, t = 0$		



Lumped Thermal Mass Phase Portrait/Anticipated Solution

ODE and Initial Condition	Fixed Points (=Steady-State Values)	Stability
$\rho c V \frac{dT}{dt} = -h A_s (T - T_\infty) + \dot{q} V$ $T_0 = 400^\circ\text{C}, t = 0$	$T_{ss} = T_\infty + \frac{\dot{q} V}{h A_s}$	Stable



~~$$\rho c V \frac{dT}{dt} = -h A_s (T - T_\infty) + \dot{q} V$$~~

$$\rightarrow T = T_\infty + \frac{\dot{q} V}{h A_s}$$