

## EE2101 Assignment 1

Sharath Chandra Sheripally (es18btech11016@iith.ac.in)

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1. Figure shows a quarter-car model commonly used for analyzing suspension systems. The car's tire is considered to act as a spring without damping, as shown. The parameters of the model are

$M_b = \text{car's body mass}$

$M_{us} = \text{wheel's mass}$

$K_a = \text{strut's spring constant}$

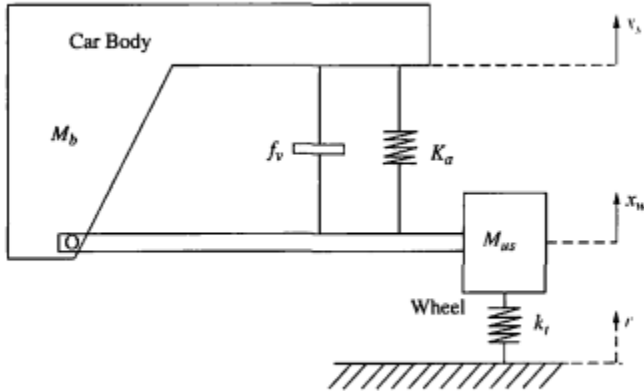
$K_t = \text{tire's spring constant}$

$f_v = \text{strut's damping constant}$

$r = \text{road disturbance (input)}$

$x_s = \text{car's vertical displacement}$

$x_w = \text{wheel's vertical displacement}$



**FIGURE P2.36** Quarter-car model used for suspension design.  
(© 1997 IEEE)

**Soln.** The two differential eqns for this system are

$$M_b \ddot{x}_s + K_a(x_s - x_w) + C_a(\dot{x}_s - \dot{x}_w) = 0$$

$$M_{us} \ddot{x}_w + K_a(x_w - x_s) + C_a(\dot{x}_w - \dot{x}_s) + K_t(x_w - r) = 0$$

Obtaining Laplace transform on both sides gives

$$(M_b s^2 + C_a s + K_a) X_s - (K_a + C_a s) X_w = 0$$

$$(M_{us} s^2 + C_a s + (K_a + K_t)) X_w - (K_a + C_a s) X_s = R K_t$$

Solving the first equation for  $X_s$  and substituting into the second one gets

$$\frac{X_w}{R}(s) = \frac{K_t(M_b s^2 + C_a s + K_a)}{(M_{us} s^2 + C_a s + (K_a + K_t))(M_b s^2 + C_a s + K_a) - (K_a + C_a s)^2}$$