

Assignment 2

Kawasaki Z650

Single-DOF vibration model
Numerical approach: pothole
(10 points)



Objective

Modal identification of the Single-DOF vibration model of the motorbike frame

From the given results of a numerical simulation, the aim of the assignment is to

- plot a time response resulting from a pothole impact;
- plot the Bode and Nyquist diagrams;
- identify the modal parameters (natural frequency and modal damping) of the 1-DOF system.

The modal parameters will be identified for the first bending mode of the motorbike frame in the vertical direction.

Simplified model of the motorcycle frame (SDOF model)

- The model simulates a constant speed motion
 - The frame is represented as a beam whose characteristics are listed in the general presentation of the project
 - Two concentrated masses are located at points B and D
 - B: represents the engine and the auxiliaries, located at mid-distance between the fork (A) and the swingarm (C) and at 250 mm below the frame
 - D: represents the driver, assumed to be on the frame
 - The frame is supported by the fork and the swingarm
 - Both are modeled as linear and rotational springs with adequate stiffness

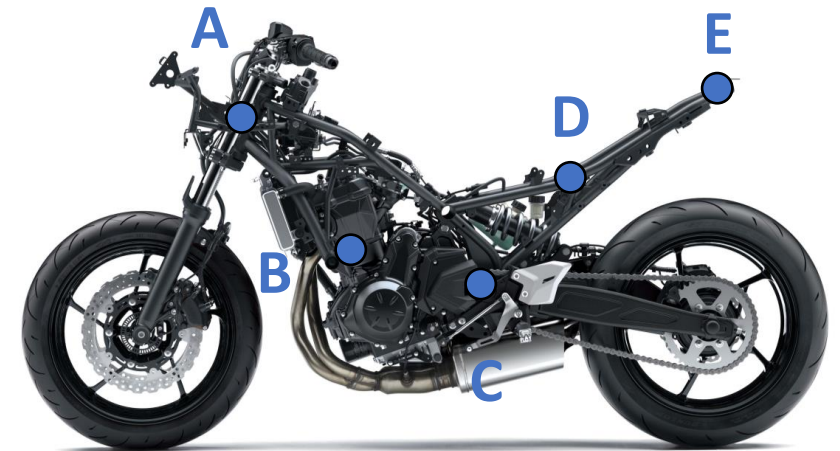
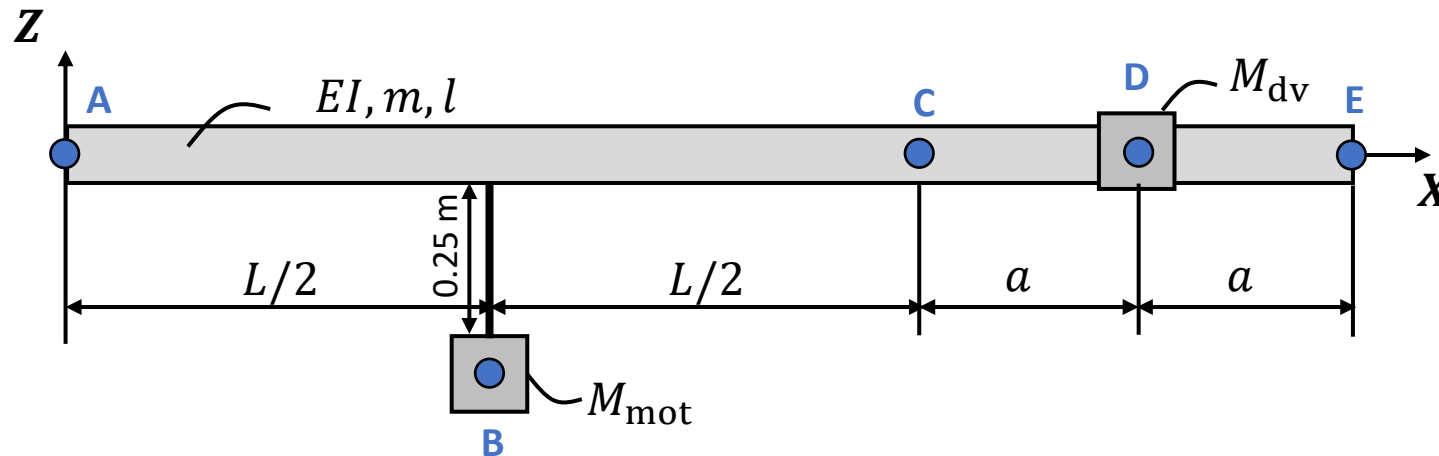


Figure 1 – SDOF model with $l = L + 2a$, $a = \frac{L}{4} = 0.2$ m.

Simplified model of the motorcycle frame

- The structure is excited by an impact simulating a pothole at the fork (A) in the vertical direction Z
- The response is measured at point A in the same direction
- For the time response, note that the time pulse signal has a duration of 0.02s (corresponding to a 20 cm long pothole at 36 km/h).

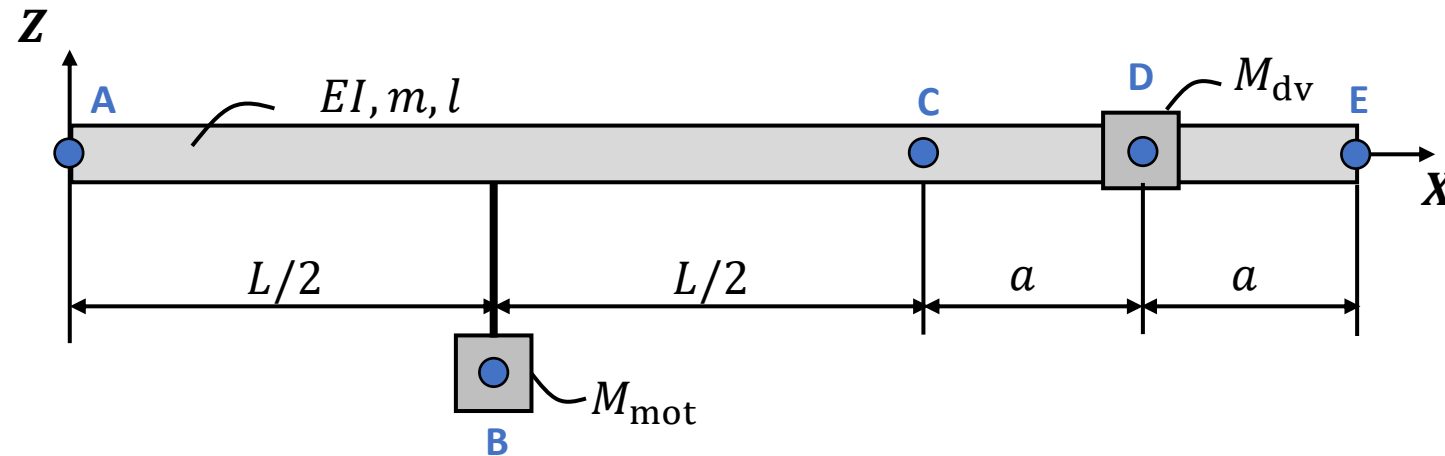


Figure 1 – SDOF model with $l = L + 2a$, $a = L/4$. Use $a = 0.2$ m.

Data available on eCampus

- The time response to the pothole impact is obtained by a numerical simulation. The time response is given in the file *P2024_irf_acc.txt*, in which two columns of data are given:

Time [s]	Acceleration [m/s ²]
...	...

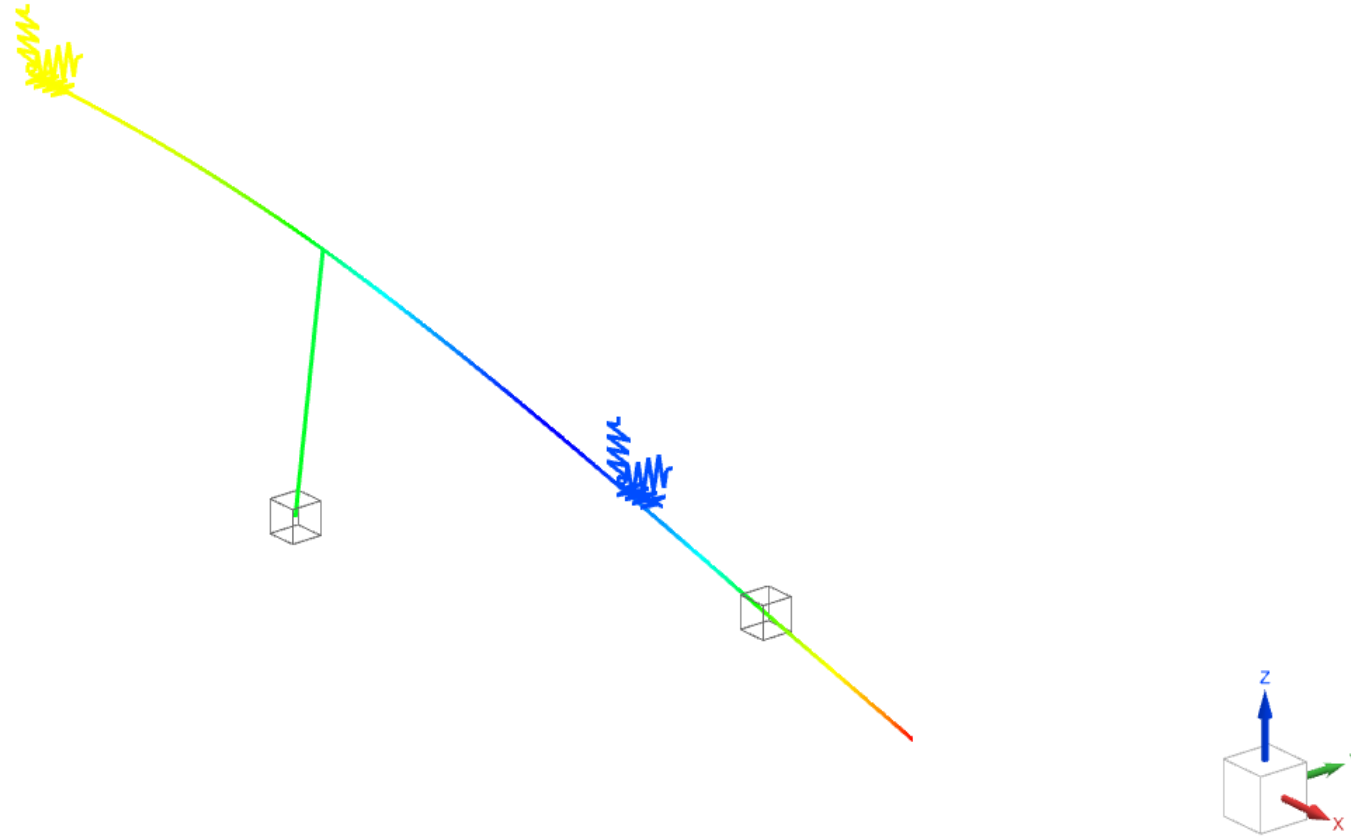
- The frequency response function (FRF) is measured in the frequency range [0 – 150] Hz. The FRF is given in the file *P2024_frf_acc.txt* in which three columns of data are present:

Frequency [Hz]	$Re(FRF)$ [m/s ² /N]	$Im(FRF)$ [m/s ² /N]
...

Project statement

- Plot the time response at point A.
 - Estimate the damped natural frequency in bending from the time data.
 - Estimate the associated damping ratio using the logarithmic decrement method.
 - Note that the time response is obtained from a time pulse signal whose duration is 0.02s.
 - What about the low damping hypothesis?
- Plot the Bode diagram and estimate the natural frequency.
 - Evaluate the quality factor and the damping ratio using the half-power method.
- Plot the Nyquist diagram and evaluate the damping ratio based on the equivalent mass of the motorbike frame system of 87.5 kg.
 - Note that this equivalent mass is estimated from the FE model of the frame system and is different from the equivalent mass found in assignment n°1 where some additional assumptions were made.
- Compare the identification methods using the time data and the frequency data with respect to their ease of use and their accuracy.

Mode 1: Bending in XZ-plane (87.49 Hz)



Specific guidelines for assignment 2

- 1) The length of the report will not exceed 5 pages including figures.
- 2) Prefer the dedicated forum on eCampus to ask questions
- 3) The deadline for the submission of the report (on eCampus platform) is fixed to
October 25, 2024 at 18:00