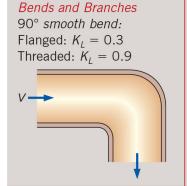
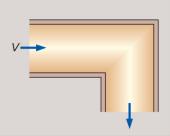


FIGURE A-27 The Moody chart for the friction factor for fully developed flow in circular pipes for use in the head loss relation  $\Delta P_L = f \frac{L}{D} \frac{\rho V^2}{2}$ . Friction factors in the turbulent flow are evaluated from the Colebrook equation  $\frac{1}{\sqrt{f}} = -2\log_{10}\left(\frac{\varepsilon/D}{3.7} + \frac{2.51}{\text{Re}\sqrt{f}}\right)$ .

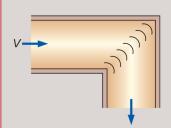
## TABLE 14-4 (CONCLUDED)



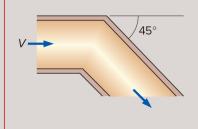
90° miter bend (without vanes):  $K_L = 1.1$ 



90° miter bend (with vanes):  $K_l = 0.2$ 



45° threaded elbow:  $K_{I} = 0.4$ 

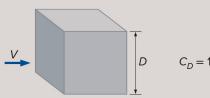


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## **TABLE 15-2**

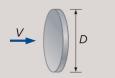
Representative drag coefficients  $\mathcal{C}_{\mathcal{D}}$  for various three-dimensional bodies based on the frontal area for Re  $> 10^4$  unless stated otherwise (for use in the drag force relation  $F_D = C_D A \rho V^2/2$  where V is the upstream velocity)

Cube,  $A = D^2$ 



 $C_D = 1.05$ 

Thin circular disk,  $A = \pi D^2/4$ 



Cone (for  $\theta = 30^{\circ}$ ),  $A = \pi D^2/4$ 



 $C_D$ 

1.1

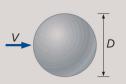
0.9

0.9

0.9

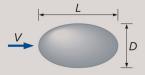
1.0

Sphere,  $A = \pi D^2/4$ 



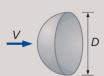
Laminar:

See Fig. 11–36 for  $C_D$  vs. Re for smooth and rough spheres. Ellipsoid,  $A = \pi D^2/4$ 

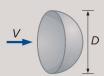


	$C_D$	
L/D	Laminar	Turbulent
	$Re \lesssim 2 \times 10^5$	$Re \gtrsim 2 \times 10^6$
0.75	0.5	0.2
1	0.5	0.2
2	0.3	0.1
4	0.3	0.1
8	0.2	0.1

Hemisphere,  $A = \pi D^2/4$ 

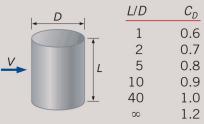


 $C_D = 0.4$ 



 $C_D = 1.2$ 

Finite cylinder, vertical, A = LD



Values are for laminar flow  $(Re \leq 2 \times 10^5)$ 

Finite cylinder, horizontal,  $A = \pi D^2/4$ 

