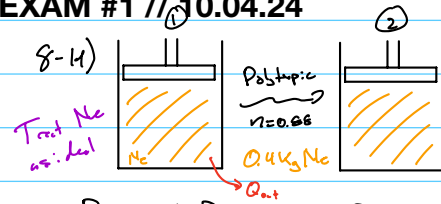


1.

Constant P



$$R = 0.4119 \frac{\text{kJ}}{\text{kgK}}$$

$$C_v = 0.6179 \frac{\text{kJ}}{\text{kgK}}$$

$$2. \text{ Water } T = 275^\circ\text{C}$$

$$S = 6 \frac{\text{kJ}}{\text{kgK}}$$

$$s_f = 3.0221$$

$$s_g = 5.8944$$

$$s_f < s_g < s$$

Superheated
Vapor

$$P_1 = 100 \text{ kPa}$$

$$P_2 = 400 \text{ kPa}$$

$$C_p = 1.0244 \frac{\text{kJ}}{\text{kgK}}$$

$$T_1 = 20^\circ\text{C} = 293.15 \text{ K}$$

$$V_2 = 0.1 \text{ m}^3$$

$$8) P_1 V_1^n = P_2 V_2^n$$

$$V_1^n = \frac{P_2 V_2^n}{P_1}$$

actually: use EOS!!

$$P_1 V_1 = m R T_1 \rightarrow V_1 = \frac{m R T_1}{P_1}$$

$$V_1 = \frac{(0.4 \text{ kg})(0.4119 \frac{\text{kJ}}{\text{kgK}})(293.15 \text{ K})}{100 \text{ kPa}} = 0.483 \text{ m}^3$$

$$9) W_{\text{piston}} = \frac{P_2 V_2 - P_1 V_1}{n} = \frac{400(0.1) - 100(0.483)}{0.88}$$

$$W_{\text{piston}} = -4.43$$

3. Heat, Work, Mass

4. Heat flows spontaneously
from higher to lower energy
m.f.f.

5. Energy is conserved

$$12) \frac{P_1 V_1}{T_1} = \frac{P_2 V_2}{T_2} \rightarrow T_2 = \frac{P_2 V_2 T_1}{P_1 V_1}$$

$$T_2 = \frac{400(0.1)(293.15 \text{ K})}{100(0.483)} = 242.77 \text{ K}$$

6. Heat Pump for a heater

Car temp: $68^\circ\text{F} (293 \text{ K}) = T_H$

Outdoor temp: $10^\circ\text{F} (26 \text{ K}) = T_L$

$$\text{COP}_{\text{HP, Carnot}} = \frac{1}{1 - \frac{T_L}{T_H}}$$

$$\text{COP}_{\text{HP}} = 0.15625$$

13) 1st Law

$$(Q_{\text{in}} - Q_{\text{out}}) + (W_{\text{in}} - W_{\text{out}}) = \dot{m}(u_2 - u_1)$$

$$-Q_{\text{out}} - W_{\text{out}} = \dot{m} \Delta u$$

7. Not possible, $\text{COP}_{\text{HP, Carnot}} > 0.15$,
and this is kJ/mK

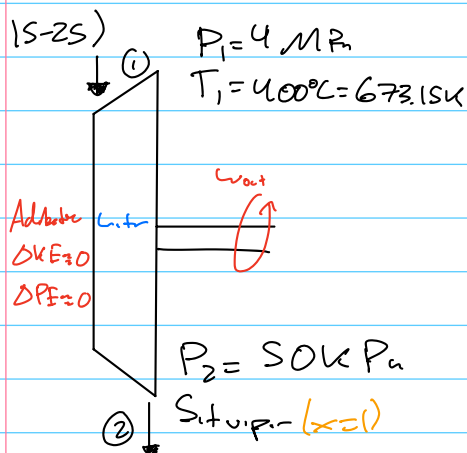
$$14) \Delta u = C_v (T_2 - T_1)$$

$$= 0.6179 \frac{\text{kJ}}{\text{kgK}} (242.77 - 293.15)$$

$$= -31.1248 \frac{\text{kJ}}{\text{kg}}$$

$$-Q_{\text{out}} - (-4.43) = 0.4 (-31.1248)$$

$$Q_{\text{out}} = 21.88 \text{ kJ}$$



15)

$$(Q_{in} - Q_{out}) (\dot{m}_{in} - \dot{m}_{out}) = \dot{m} \left[(h_2 - h_1) + \frac{1}{2} (v_2^2 - v_1^2) + g \Delta z \right]$$

$$\dot{m}_{in} - \dot{m}_{out} = \dot{m} (h_2 - h_1)$$

$$\dot{W}_{out} = \dot{m} (h_2 - h_1)$$

16-17) $P_1 = 4 \text{ MPa}$ } $\xrightarrow{\text{sat}}$ $h_1 = 3214.5 \frac{\text{kJ}}{\text{kg}}$
 $T_1 = 400^\circ\text{C}$ } $s_1 = 6.7714 \frac{\text{kJ}}{\text{kg K}}$

16 $h_1 = 3214.5 \frac{\text{kJ}}{\text{kg}}$

17 $s_1 = 6.7714 \frac{\text{kJ}}{\text{kg K}}$

18-19) $P_2 = 50 \text{ kPa}$ } $\xrightarrow{\text{sat}}$ $h_{2A} = 2645.2 \frac{\text{kJ}}{\text{kg}}$
 $x=1$ } $\xrightarrow{\text{mix}}$ $s_{2A} = 7.5931 \frac{\text{kJ}}{\text{kg K}}$

20) Reversible \rightarrow ideal \rightarrow isentropic

Phase change fluid: $s_2 = s_1$

$$s_{2s} = 6.7714 \frac{\text{kJ}}{\text{kg K}}$$

21) Ideal: $x = 0.874$

$$h = h_f + x(h_g - h_f)$$

$$h_{2s} = 2354.81 \frac{\text{kJ}}{\text{kg}}$$

50 kPa } $h_f = 340.54$
 $x = 0.874$ } $h_g = 2645.2$

22) $\eta_T = \frac{h_1 - h_{2A}}{h_1 - h_{2s}} = \frac{3214.5 - 2645.2}{3214.5 - 2354.81} = 0.662$

23) $\Delta s_{s,s} = s_{2A} - s_{1A} = 7.5931 - 6.7714 = 0.8217$

24) Adiabatic \rightarrow no heat exchanged so $\Delta s_{sur} = 0$ $\frac{Q}{T}$

25) $\Delta s_{tot} = \Delta s_{s,s} + \Delta s_{sur} = 0.8217$