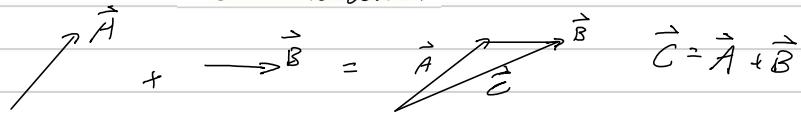


Vectors and 2D kinematics

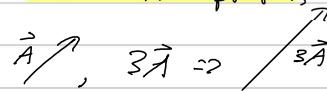
Recall: Vectors have both magnitude and direction.

Vector Addition

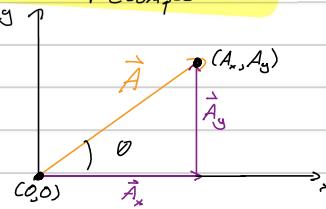


Length = "magnitude" $|\vec{A}|$

Scalar Multiplication



Vector Decomposition



Vectors are defined by magnitude!

$|\vec{A}| = \text{magnitude} = A_{\text{mag}} = \sqrt{A_x^2 + A_y^2}$
Direction = θ (using reference angle)

Or we can break it into components!
 $\vec{A} = \vec{A}_x + \vec{A}_y$

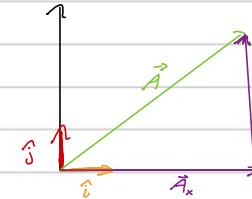
To find A : f A_x, A_y given: $A = \sqrt{A_x^2 + A_y^2}$ using
To find θ : f A_x, A_y given: $\theta = \arctan\left(\frac{A_y}{A_x}\right)$ Crazy

Cartesian Basis Unit Vector



Don't use $< >$
Notation is too loss!

Unit Vector = length of one!



Means...

$$\begin{aligned}\vec{A}_x &= A_x \hat{i} \\ \vec{A}_y &= A_y \hat{j}\end{aligned}$$

$$\therefore \vec{A} = A_x \hat{i} + A_y \hat{j}$$

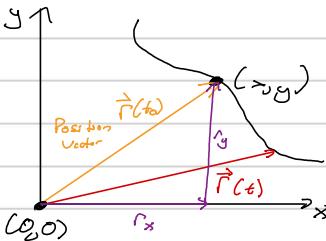
Vector Representation

Example: $\vec{C} = \vec{A} + \vec{B}$

$$\begin{cases} \vec{A} = A_x \hat{i} + A_y \hat{j} \\ \vec{B} = B_x \hat{i} + B_y \hat{j} \end{cases} \therefore \vec{C} = (A_x + B_x) \hat{i} + (A_y + B_y) \hat{j}$$

Path Integrals will Come Later

Kinematics w/ Vectors



$$\vec{r} = \vec{r}_x + \vec{r}_y = r_x \hat{i} + r_y \hat{j}$$

Position Vectors = v-vectors
whose components are its
own coordinates

$$\vec{r} \equiv x \hat{i} + y \hat{j}$$

$$\Rightarrow \vec{v} = \frac{d\vec{r}}{dt} \Rightarrow \vec{a} = \frac{d\vec{v}}{dt}$$

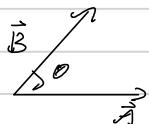
$$\begin{aligned}\vec{v} &= \frac{dx}{dt} \hat{i} + \frac{dy}{dt} \hat{j} \\ \vec{a} &= \frac{d^2x}{dt^2} \hat{i} + \frac{d^2y}{dt^2} \hat{j}\end{aligned}$$

$$\vec{a} = a_x \hat{i} + a_y \hat{j}$$

Example: $\vec{r}(t) = (Bt^2) \hat{i} + (Ce^{-\frac{t}{\tau}}) \hat{j}$ t, B, C Given Constants Find Velocity and accel

$$\vec{v}(t) = (2Bt) \hat{i} - \left(\frac{C}{\tau} e^{-\frac{t}{\tau}}\right) \hat{j} \quad \text{and} \quad \vec{a}(t) = (2B) \hat{i} - \left(\frac{C}{\tau^2} e^{-\frac{t}{\tau}}\right) \hat{j}$$

Dot Products!



$$C = \vec{A} \cdot \vec{B} = AB \cos \theta$$

$$\hat{i} \cdot \hat{i} = 1, \hat{i} \cdot \hat{j} = 0$$

Using Decomposition

$$\therefore C = (A_x \hat{i} + A_y \hat{j}) \cdot (B_x \hat{i} + B_y \hat{j})$$

$$\text{FOLV} = A_x B_x (1) + A_x B_y (0) + A_y B_x (0) + A_y B_y (1)$$

$$C = A_x B_x + A_y B_y$$

Example: Related to HW4

Define: $\vec{r}(t) = R \cos(\omega t) \hat{i} + R \sin(\omega t) \hat{j}$, R, ω given

$$|\vec{r}| = \sqrt{x^2 + y^2} \rightarrow |\vec{r}| = \sqrt{(R \cos(\omega t))^2 + (R \sin(\omega t))^2}$$

Distance Magntude

Suppose



$$= \sqrt{R^2 (\cos^2(\omega t) + \sin^2(\omega t))} = \sqrt{R^2} = R$$

$$\text{Speed} = |\vec{v}| = \sqrt{v_x^2 + v_y^2}$$

$$\vec{v} = \frac{d\vec{r}}{dt} = -\omega R \sin(\omega t) \hat{i} + \omega R \cos(\omega t) \hat{j}$$

$$\therefore |\vec{v}| = \sqrt{R^2 \omega^2 (\sin^2(\omega t) + \cos^2(\omega t))} = R\omega$$

Omg! its the radial constant

Torce Derivatives of Functions of time!

$$\vec{a} = -R\omega^2 \cos(\omega t) \hat{i} - R\omega^2 \sin(\omega t) \hat{j}$$

$$|\vec{a}| = \sqrt{R^2 \omega^4 (\cos^2(\omega t) + \sin^2(\omega t))}$$

$$= \sqrt{R^2 \omega^4} = R\omega^2 = \frac{v^2}{R}$$

$$a = \frac{v^2}{R} = a_c$$

a_c is centripetal acceleration

Projectile Motion

x -direction:

No air resistance \rightarrow velocity is constant

$$a_x = 0$$

$$v_{x_0} = \text{Const} = v_{x_0}$$

$$x = x_0 + v_{x_0} t$$

y -direction:

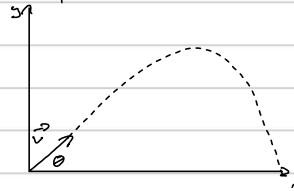
Free Fall

$$a_y = -g$$

$$V_y = V_{y_0} - gt$$

$$y = y_0 + V_{y_0} t - \frac{1}{2} g t^2$$

Graph



We see: $v_{x_0} = v_0 \cos \theta$ and $v_{y_0} = v_0 \sin \theta$

$$\therefore \vec{r} = x \hat{i} + y \hat{j}$$

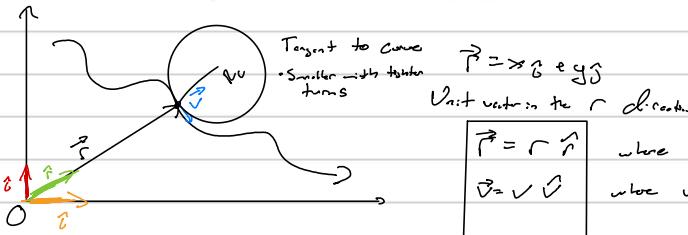
$$\text{In order to } \vec{r}(t) = (x_0 + v_{x_0} t) \hat{i} + (y_0 + v_{y_0} t - \frac{1}{2} g t^2) \hat{j}$$

$$\text{Position shown above} \quad \vec{r}(t) = (v_{x_0} t) \hat{i} + (v_{y_0} t - \frac{1}{2} g t^2) \hat{j}$$

$$\vec{a}(t) = 0 \hat{i} - g \hat{j}$$

Motion of Projectile in vector form

Example: A particle on a given path



$$\vec{r} = r \hat{r}$$

$$\vec{v} = v \hat{v}$$

where $r = |\vec{r}|$ and $|\hat{r}|=1$ r is mag, \hat{r} is dir

where $v = |\vec{v}|$ and $|\hat{v}|=1$ v is mag, \hat{v} is dir

$$\vec{a} = a_x \hat{i} + a_y \hat{j}$$

$$\vec{a} = a_r \hat{r} + a_\theta \hat{\theta}$$

Tangential Component Centripetal Component

R_c is radius of curvature

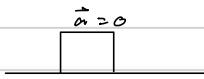
\hat{c} is \hat{r} to $\hat{\theta}$

$$\text{magnitude } a_r = \frac{dv}{dt} (\text{v}) \quad \text{vloc w/ respect to speed}$$

$$a_c = \frac{v^2}{R_c}$$

Both depend on speed

Forces at Angles



$$F_{\text{Net}} = 0$$



$$F_{\text{Net}} = 0$$

Steps

- (1) FBD
- (2) Coordinate System
- (3) Kinematic Constraint
- (4) NZL

(1.1) Decompose Forces

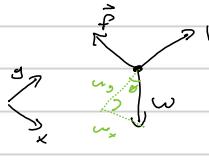
Decomposing necessary forces into their necessary components is very important!

Suppose there is friction

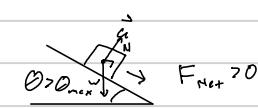


Given m, θ ; Find (i) all forces
 $\alpha=0$ as others?

FBD Block



$$\begin{aligned} & \text{N: } F_x = m a_x \quad F_y = m a_y \\ & \text{w: } w_x = w \sin \theta = m g \sin \theta \quad w_y = w \cos \theta = m g \cos \theta \\ & \text{f: } f_x = f \quad f_y = 0 \\ & \text{Friction: } w_x - f = 0 \quad N - w_y = 0 \\ & \text{Sum of forces: } m g \sin \theta - f = 0 \quad N - m g \cos \theta = 0 \\ & \text{Resultant force: } f = m g \sin \theta \quad N = m g \cos \theta \end{aligned}$$



$\theta > \theta_{\text{max}}$ the maximum angle required for the body to remain at rest.

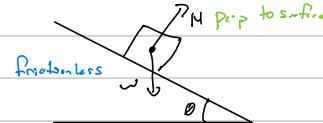
Given m, θ ; find (i) all forces

PBD Block

Kinematic Constraint

$$a_x = a$$

$$a_y = 0$$



Frictionless

$$F_{\text{Net}} = m a$$

$F_y = m a_y$ ← Normal Force and y component of the weight force

$$N - w_y = m a_y$$

$$N - w \cos \theta = 0$$

$$N \sin \theta = m a$$

$$N = m g \cos \theta$$

NZL

$$F_{\text{Net}} = m a$$

$$w_x = m a_x \quad \text{only } w_x \text{ in } x\text{-dir}$$

$$w_{y\text{ or }x} = m a$$

$$m g \sin \theta = m a \Rightarrow a = g \sin \theta$$

Small $\theta \rightarrow$ Small a
Large $\theta \rightarrow$ Large a

Suppose there is Kinetic Friction

Given m_k, m, θ find (i) a
(ii) all forces

NZL

$$N: F_y = m a_y$$

Same as fr

b/t to fr left

$$N = m g \cos \theta$$

$$\therefore w_x - f = m a_x$$

$$m g \sin \theta - f = m a$$

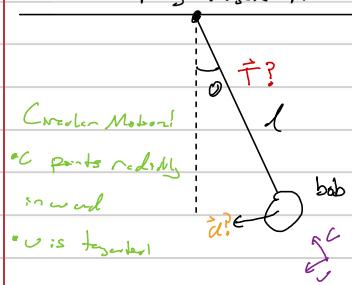
$$m g \sin \theta - m_k m g \cos \theta = m a$$

$$m g (\sin \theta - m_k \cos \theta) = m a$$

$$a = g(\sin \theta - m_k \cos \theta)$$

Pendulums

Jumping right in with an example



Pendulum bob with mass m released at rest angle θ

(Q1)

What is the tension on string?

(Q2)

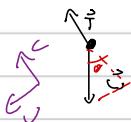
What is the acceleration of bob immediately after its released?

Given mass m , θ , and length of string l

a_c and a_v must be considered

FBD Bob

N2L : c-coord



$$F_c = m a_c$$

$$F_c = m \left(\frac{v^2}{l}\right) = m \left(\frac{0^2}{l}\right) = 0$$

$$T - w \cos \theta = 0 \quad (1)$$

$$T - mg \cos \theta = 0 \rightarrow T = mg \cos \theta \quad (2)$$

N2L : v-coord

$$F_v = m a_v$$

$$w \sin \theta = m a_v$$

$$m g \sin \theta = m a_v$$

$$a_v = g \sin \theta$$

(Q3) What is the tension on string when bob at lowest position?



Still axis coordinate
as path is still circular

FBD Bob

N2L v-coord

$$F_c = m a_c$$

$$T - w = m \left(\frac{v^2}{l}\right)$$

$$T - mg = m \left(\frac{v^2}{l}\right) \text{ stop!}$$

v is unknown here!

F-dia

Conserv in conservative

\vec{T} does no work $\Rightarrow \vec{T}$ top path
 $B_{\text{rel}} = A_{\text{rel}}$ ("B_{rel}" = release

$E_{\text{pot}} > E_{\text{kin}}$ ("A_{rel}" = lowest position)

$U + K = U' + K'$ Condition is met!

$$\Rightarrow T_{\text{mg}} = \frac{mv^2}{l} \text{ and } v^2 = \sqrt{2gl(1-\cos\theta)}$$

$$T = mg + \frac{m(2gl(1-\cos\theta))}{l}$$

$$T = mg + 2mg(1-\cos\theta)$$

$$T = mg(1+2(1-\cos\theta))$$

$$T = mg(1+2-2\cos\theta) \rightarrow T = mg(3-2\cos\theta)$$



$$mg y + \frac{1}{2}mv^2 = mg y + \frac{1}{2}mv^2$$

$$y = -l \cos \theta \quad , y = -l$$

$$V = 0 \quad , v^2 = ?$$

$$mg(-l \cos \theta) + \frac{1}{2}mv^2 = mg(-l) + \frac{1}{2}mv^2$$

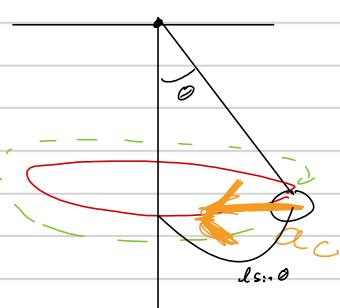
$$-mg l \cos \theta = -mg l + \frac{1}{2}mv^2$$

$$gl - gl \cos \theta = \frac{1}{2} v^2$$

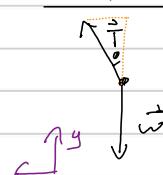
$$\sqrt{v^2} = \sqrt{2gl(1-\cos\theta)}$$

$$\therefore \sqrt{v^2} = \sqrt{2gl(1-\cos\theta)}$$

Central Pendulums



FBD Bob



N2L y-coord

$$F_y = m a_y \quad a_y = 0$$

$$T \cos \theta - w = 0$$

$$\boxed{T = \frac{mg}{\cos \theta}}$$

N2L c-coord

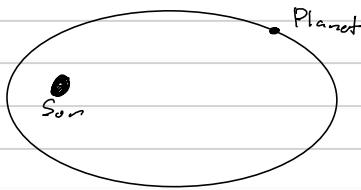
$$F_c = m a_c$$

$$T \sin \theta = m \frac{v^2}{l \sin \theta}$$

$$\frac{mg \sin \theta}{\cos \theta} = \frac{mv^2}{l \sin \theta}$$

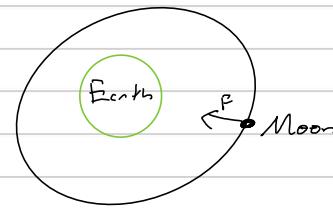
$mg \tan \theta = \frac{mv^2}{l \sin \theta}$, solve for v^2

Planets!



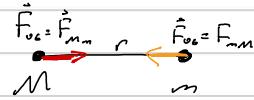
Kepler's 3rd Law (K3L)

$$\frac{P^2}{R^3} = \frac{(Period)^2}{("radius")^3} = \text{constant?}$$



Universal Gravity

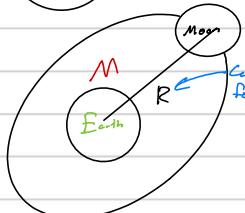
$$F_{\text{Grav}} = \frac{G \cdot M \cdot m}{r^2}$$



G = Gravitational Constant

$$= 6.67 \times 10^{-11} \text{ N} \cdot \text{m}^2 \cdot \text{kg}^{-2}$$

we only note it b/c of earth's size
it's ~~good~~ miss



Situation $M \gg m$

Orbit \approx Circular

NZL Moon

$$F_{\text{Moon}} = m \cdot a_c$$

$$\frac{G \cdot M_{\text{Earth}}}{R^2} = m \cdot \frac{v^2}{R}$$

$$\text{Solve for } \frac{P^2}{R^3}$$

(1) what are the forces on the moon

$$\rightarrow \text{Period} = \frac{2\pi R}{v} = P$$

displace
time

K3L back!

$$1 \text{ kg} \quad 1 \text{ m} \quad 1 \text{ kg} \quad F_{\text{Grav}} = \frac{G(1)(1)}{1} = G \quad \text{b/c anything out will}$$

where "g" comes from

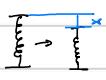
$$g_E = \frac{G M_E}{R_E^2}$$

Force Menu

Symbol	what	Known?	Conservative?	Contact?
\vec{w}	weight	$w = mg$	Yes	No
\vec{N}	Normal Push	No	No	Yes
\vec{T}	Tension Pull	No	No	Yes
\vec{F}	Friction	Sometimes	Absolutely No!	Yes ← known if kinetic friction unk
F_{Grav}	Universal Gravity	Yes	Yes	No
F_{sp}				

Hooke and Springs

"Spring"



$$F_{\text{sp}} = k|x| \quad x \text{ is displacement from about natural spring length}$$

Compressed or stretched
 x is the change in length

$$F_{\text{sp}} = k|x|$$

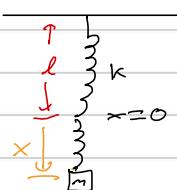
k = spring constant

x = displacement relative to natural spring

large k equates to increased force

with equal displacement compresses $-k$

Example



$x=0$

F_{BD}

$x < 0$

$x > 0$

$x = 0$

NZL x

$$F_x = m a_x \rightarrow 0$$

$$w = mg$$

$$w - F_{\text{sp}} = 0$$

$$F_{\text{sp}} =$$

$$mg - kx = 0$$

$$F_{\text{sp}} =$$

$$x = \frac{mg}{k}$$

You can only
assume $a=0$
when mass
or object is
in equilibrium

Definition of Potential Energy

$$U_{\vec{F}}(P) - U_{ref}(P_{ref}) = - \int_{P_{ref}}^P \vec{F} \cdot d\vec{r}$$

From position P_{ref} to P . $\vec{F} \cdot d\vec{r} = \text{Force} \theta$
Change w/ time

$$\text{Work done by } \vec{F} = -\Delta U_F$$

work done by
conservative force

Path integral

Computing the integral

Weight \vec{w} and P_g

Pot. Energy

$$U_{\vec{w}}(y) - U_{ref}(y_{ref}) = - \int_{y_{ref}}^y \vec{w} \cdot d\vec{r}$$

3 for weight

$$U_{\vec{w}}(y) - U_{ref}(y_{ref}) = - \int_{y_{ref}}^y \vec{w} \cdot d\vec{r} = mg dy \cos(180^\circ)$$

$$U_{\vec{w}}(y) - U_{ref}(y_{ref}=0) = - \int_{y=0}^y (-mg dy) = mg dy$$

Set up P to 0 in

Physics, not always

true in PHYS122

$$U_{\vec{w}}(y) = mgy$$

Just an easy integral

Calculating for Spring Force

$$U_{sp}(x) - U_{ref}(x_{ref}) = - \int_{x_{ref}}^x \vec{F}_{sp} \cdot d\vec{r}$$

$$U_{sp}(x) - U_{ref}(x=0) = - \int_{x=0}^{x=x} (-K_x dx)$$

$$U_{sp}(x) = \frac{1}{2} K_x x^2$$

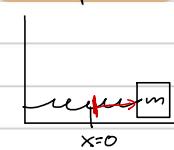
$$\vec{F}_{sp} = K_x \frac{d\vec{r}}{dx}$$

$$\vec{F}_{sp} \cdot d\vec{r} = K_x dx \cos(180^\circ)$$

$$|\vec{F}_{sp}| = -K_x dx$$

Spring Free arguments

Example



Release at rest

at $x = x_i$

Find max speed

Given

K, m, x_i

Conserv C of ME when asked to find speed

All conservative are one word

"Before" = "After"

$$U + K = U' + K'$$

$$\frac{1}{2} K x_i^2 + \frac{1}{2} m v_i^2 = \frac{1}{2} K x^2 + \frac{1}{2} m v^2$$

$$x = x_i, v = 0, x' = 0 \quad v' = v_p ?$$

$$\frac{1}{2} K x_i^2 + 0 = 0 + \frac{1}{2} m v_p^2$$

$$\therefore v_p = x_i \sqrt{\frac{K}{m}}$$

Computing for Universal Gravity

$$U_{\vec{F}_G}(r) - U_{ref}(r_{ref}) = - \int_{r_{ref}}^r \vec{F}_{\vec{F}_G} \cdot d\vec{r}$$

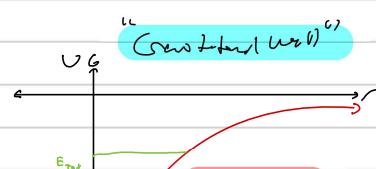
From r to r'

$$\vec{F}_{\vec{F}_G} \cdot d\vec{r} = \frac{GM_m}{r^2} dr \cos(180^\circ)$$

For goes to 0
when $r = \infty$

$$U_{\vec{F}_G}(r) - U_{ref}(r_{ref} = \infty) = - \int_{r=\infty}^{r=r} \left(-\frac{GM_m}{r^2} \right) dr$$

$$U_{\vec{F}_G}(r) = -\frac{GM_m}{r}$$



Conservative Forces

Weight \vec{w} $U_{\vec{w}} = mgy$

Spring Force $U_{sp} = \frac{1}{2} K x^2$

Universal Grav. $U_G = -\frac{GM_m}{r}$

C. of FME

$$E_{\text{Tot}} = E'_{\text{Tot}}$$

$$U + K = U' + K'$$

U' can be a sum of different potential energies

Escape Velocity

ignores Air resistance, high R sends an object to f atmospheric as $\frac{1}{r^2}$ decreases



At $r = \infty$ C. of FME

$$U + K = U' + K'$$

$$-\frac{GM_m}{r} + \frac{1}{2} mv^2 = -\frac{GM_m}{r'} + \frac{1}{2} mv'^2$$

$$-\frac{GM_m}{r} + \frac{1}{2} mv^2_{esc} = 0 + 0$$

$$\text{Algebra} \Rightarrow v_{esc} = \sqrt{\frac{2GM_m}{r}}$$

$$\approx 1.2 \times 10^4 \text{ m/s}$$

or 7 miles/sec

Thrown with some velocity
v_esc. Lat. radius = R. Masses
of object not needed. Plan mass m.
thrown tangential. If it lands
is $\theta = \pi$ and $r = \infty$. Find escape
velocity v

More on Systems

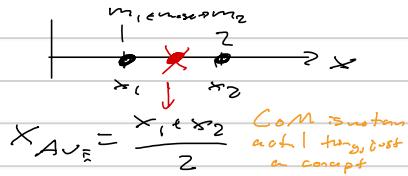
NCL for Systems

$$\vec{F}_{\text{net}} = M_{\text{sys}} \vec{a}_{\text{sys}}$$

Ext

works for systems of
n objects and directions
only with COM

Center of Mass (Com)



$$x_{\text{com}} = \frac{x_1 + x_2}{2}$$

COM is just a concept

Suppose $m_1 \neq m_2$, find weighted Avg

2 pts: $x_{\text{com}} = \frac{m_1 x_1 + m_2 x_2}{m_1 + m_2}$

weighted avg eqn

N pts: $x_{\text{com}} = \frac{m_1 x_1 + m_2 x_2 + \dots + m_n x_n}{m_1 + m_2 + \dots + m_n}$

$y_{\text{com}} = \frac{m_1 y_1 + m_2 y_2 + \dots + m_n y_n}{m_1 + m_2 + \dots + m_n}$

2 coordinates
with n bodies
with given masses
and positions

Time derivatives of Com

$$\frac{d x_{\text{cm}}}{dt} = v_{\text{cm}} = \frac{m_1 v_1 + m_2 v_2}{m_1 + m_2}$$

derivative of a sum is
the sum of derivatives

$$\frac{d v_{\text{cm}}}{dt} = a_{\text{cm}} = \frac{m_1 a_1 + m_2 a_2}{m_1 + m_2}$$

can be expanded to N points

NCL Systems

$$\vec{F}_{\text{net}} = M_{\text{sys}} \vec{a}_{\text{cm}}$$

vector be it
applies to N
number of bodies

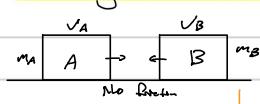
If $\vec{F}_{\text{net}} = 0 \Rightarrow$ "isolated" $\Rightarrow a_{\text{cm}} = 0$

$v_{\text{cm}} = \text{constant}$

$M_{\text{tot}} \cdot v_{\text{cm}} = P_{\text{tot}}$ COM Momentum!

Inertial reference frame is
defined to be at rest
moving with constant velocity

Total Elasto Collisions



$K = K'$ is w/ total elasto means

$$\therefore \frac{1}{2} m_A v_A^2 + \frac{1}{2} m_B v_B^2 = \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B v_B'^2$$

↳ $P_{\text{tot}} = P_{\text{tot}'}$

$m_A v_A + m_B v_B = m_A v_A' + m_B v_B'$

Conservation of Momentum

Conservation of Energy

Analyze and a system of two

to solve for quadratically

use two method!

Bob Brown Method

$$v_A - v_B = v_B' - v_A'$$

Only true in Total Elasto Collisions

Must be true!!!

Center of Mass Reference Frame

$$v_{\text{cm}} = \frac{m_A v_A + m_B v_B}{m_A + m_B} = \dots$$

can be calculated: P
 m_A, m_B, v_A , and v_B are
constants that move!

Relate to C.M Reference Frame $v_{\text{cm}} = 0$

Choose coordinate system
that moves with center of
mass (Com) of system!

$$v_{\text{cm}} = 0 \quad \text{if}$$

Example: $m_A = m_B = m$

$$v_A = v_0 \quad v_B = 0$$

$$(1) \quad m v_0 = m v_A' + m v_B'$$

$$(2) \quad v_0 = v_B' - v_A'$$

$$v_B' = v_0 + v_A'$$

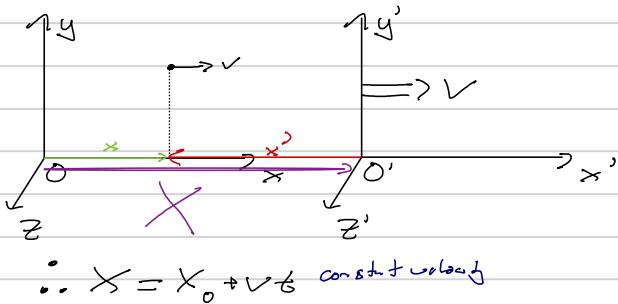
into (1)

$$m v_0 = m v_0 + m (v_0 + v_A')$$

$$2 v_0' = 0 \Rightarrow v_0' = 0$$

$$(2) \rightarrow v_B' = v_0$$

Reference Frames



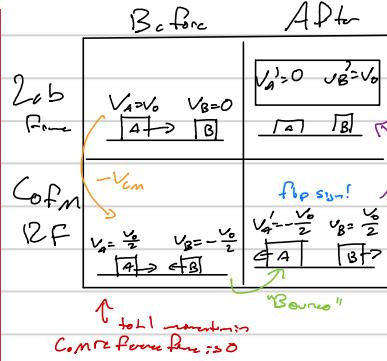
$$\begin{aligned} \bar{x} &= x - X \\ \bar{x} &= x - (x_0 + vt) \\ \bar{v} &= v - V \end{aligned}$$

$\bullet \Delta t = \text{stop time}$

\bullet Center-of-mass Ref. Frame

$$V_{\text{cm}} = V_{\text{lab}} - V_{\text{cm}}$$

Collisions 4-box method



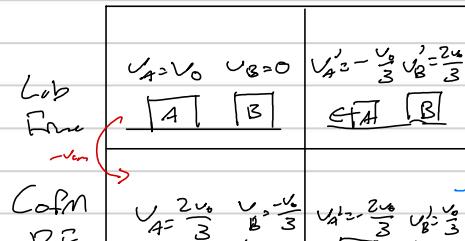
$$\begin{aligned} V_{\text{cm}} &= \frac{m_A V_0 + m_B V_0}{m_A + m_B} \\ &= \frac{m_A V_0}{m_A + m_B} \quad m_A = m_B = m \\ V_{\text{cm}} &= \frac{m V_0}{m+m} = \frac{V_0}{2} \end{aligned}$$

$$\begin{aligned} V_A' &= 0 \\ V_B' &= V_0 \end{aligned}$$

Example: $m_A = m$ $m_B = 2m$

$$V_A = V_0 \quad V_B = 0$$

B_c Frame A' Frame

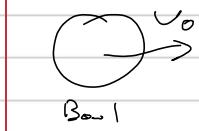


$$V_{\text{cm}} = \frac{m V_0}{m+2m}$$

$$\begin{aligned} V_A' &= -\frac{V_0}{3} \\ V_B' &= \frac{2V_0}{3} \end{aligned}$$

0 momentum total Bounce!

Example: Total elastic with rotating guns



Estimate V_{cm} !

Bowl

Golf

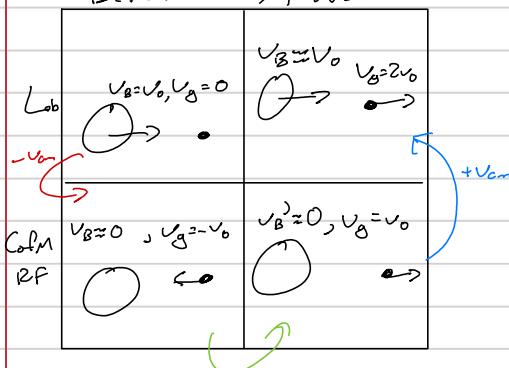
Let Bowl mass = m_B

or trust Let Golf mass = m_G

Mass Bigger! $\rightarrow m_B \gg m_G$

B_c Frame

A' Frame



$$V_{\text{cm}} = \frac{m_B V_0 + 0}{m_B + m_G}$$

$$\approx \frac{m_B V_0}{m_B + 0} = V_0$$

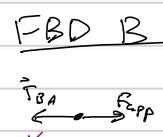
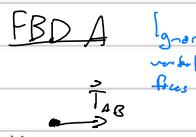
If one mass much greater, assume it doesn't move V_{cm}

Solution principle



Frictionless

Final Tension



Example: Gun $m_A = m_B = m$ and $F_{\text{app}} = \text{constant}$ \leftarrow $F_{\text{app}} \perp \text{to gun in 1D}$
 result of \downarrow $v_{\text{rel}} = \text{zero}$

Kinematic Constant
 $a_A = a_B = a$

Shortcut!

FBD System

$$\sum F_x = P_{\text{app}}$$

N2L x-dir System (A+B)

$$\begin{aligned} F_{x,\text{sys}} &= m_{\text{sys}} a_{\text{sys}} \\ F_{\text{app}} &= 2m a \end{aligned}$$

$$a = \frac{F_{\text{app}}}{2m}$$

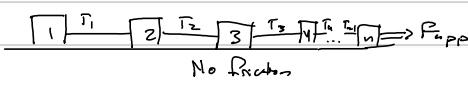
N2L A + dir

$$\begin{aligned} F_x &= m_a a \\ T_A &= m \left(\frac{F_{\text{app}}}{2m} \right) \end{aligned}$$

$$\therefore T = \frac{F_{\text{app}}}{2}$$

Calculated already

Example n number of blocks



Given: m_i , N , F_{app} . Find T_i

N2L new S_{distr}

$$F_{sys} = M_{sys} a_{sys}$$

$$T_i = (im) \frac{F_{app}}{N}$$

$$\therefore T_i = \frac{i}{N} F_{app}$$

$$a_1 = a_2 = a = a_{sys}?$$

PBD S_{distr} all blocks

To find T_i , do Rm as a system

the boundary of the system is at the point

where that force is applied

Suppose $i=3$

$$\rightarrow F_{app}$$

N2L \rightarrow dr S_{distr}

$$F_{sys} = M_{sys} a_{sys}$$

$$F_{app} = (Nm)a$$

$$a = \frac{F_{app}}{N}$$

PBD new S_{distr}

New S_{distr} = leftmost i blocks

PBD new S_{distr}

$\rightarrow T_i$ \leftarrow only friction PBD S_{distr}

once unit! δ

Example: Massive 120m

Massive Rope length l , F_{app} given

$$0111111111 \rightarrow F_{app}$$

Fraction loss

$T(x)$: Tension in rope at distance x from left end

FBD Rope

N2L 120m S_{distr}

$$F_{sys} = m_{sys} a$$

$$F_{app} = m a$$

$$a = \frac{F_{app}}{m}$$

New S_{distr} $T(x)$ \rightarrow F_{app} \rightarrow

left side up to x

FBD

N2L new S_{distr}

$$F_{sys} = m_{sys} a$$

$$T(x) = (\frac{x}{120}) a$$

$$T(x) = (\frac{x}{120}) (\frac{F_{app}}{m})$$

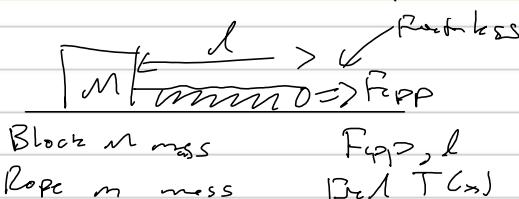
ratio of rope
length times tension

$T(x)$ \rightarrow F_{app}

$$\rightarrow T(x)$$

$$\therefore T(x) = (\frac{x}{120}) F_{app}$$

Example: Massive rope pulls block



N2L w/ hole

$$F_{sys} = M_{sys} a$$

$$F_{app} = (M+m) a_{sys}$$

$$a_{sys} = \frac{F_{app}}{m+m}$$

New S_{distr} = Boundary @ x

FBD

N2L new S_{distr}

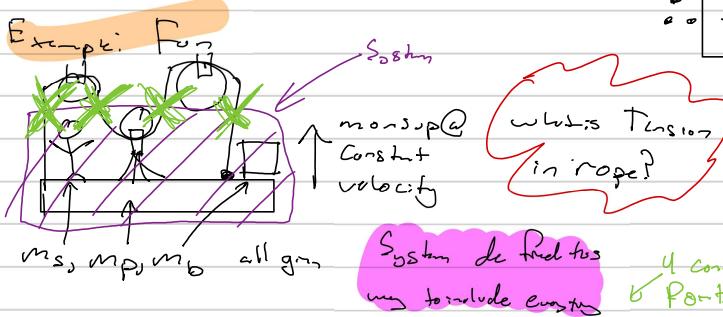
$$F_{new} = M_{new} a_{sys}$$

$$T(x) = ((\frac{x}{l}) m + M) (\frac{F_{app}}{m+m})$$

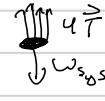
simplifying

$$\therefore T(x) = \left(M + \frac{x}{l} m \right) \frac{F_{app}}{m+m}$$

$$T(x) = \left(\frac{lm}{l+x} + M \right) \frac{F_{app}}{m+m}$$



FBD S_{distr}



N2L S_{distr}

$$F_{sys} = m_{sys} a_{sys}$$

$$4T - M_{sys} g = 0$$

$$T = \frac{M_{sys} g}{4}$$

$$\bar{T} = \frac{(m_s + m_p + m_b)g}{4}$$