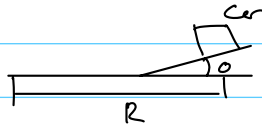
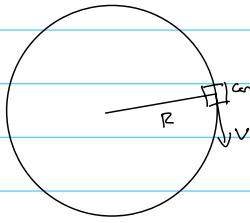


## Practice Exam 1

1.



Given

- Car Mass  $M$
- Radius of Path  $R$
- $\theta$  not given

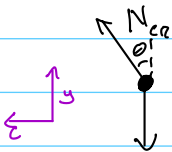
a) UCM Circular Path with Radius  $R$ . Given Speed  $v$

$$a = \frac{v^2}{R}$$

b) NZL  $F_{\text{Net}} = m a$   
 $a = \frac{v^2}{R}$   $m = M$

$$F_{\text{Net}} = M \frac{v^2}{R}$$

c) FBD Car d)  $N_c = N \sin \theta$  NZL  $\leftarrow$  dir  
 $N_g = N \cos \theta$   $F_c = m a_c$



$$N \sin \theta = M \frac{v^2}{R}$$

NZL  $y$ -dir  
 $F_g = m a_y$

$$N \cos \theta - mg = 0$$

$$N \cos \theta = mg$$

$$N = \frac{mg}{\cos \theta}$$

$$\frac{\sin \theta}{\cos \theta} M_b = M \frac{v^2}{R}$$

$$\tan \theta = \frac{v^2}{g R}$$

$$\theta = \arctan \left( \frac{v^2}{g R} \right)$$

$$2. x(t) = R\omega t - C\sin(\omega t)$$

$$y(t) = R - C\cos(\omega t)$$

$$a) \vec{r}(t) = (R\omega t - C\sin(\omega t))\hat{i} + (R - C\cos(\omega t))\hat{j}$$

$$\begin{aligned}\vec{r}\left(\frac{\pi}{\omega}\right) &= (R\omega\left(\frac{\pi}{\omega}\right) - C\sin(\pi))\hat{i} + (R - C\cos(\pi))\hat{j} \\ &= (R\pi)\hat{i} + (R+C)\hat{j}\end{aligned}$$

$$\boxed{\vec{r}(t) = (R\pi)\hat{i} + (R+C)\hat{j}}$$

$$b) \vec{v}(t) \equiv \frac{d}{dt} \vec{r}(t) = (R\omega - C\omega\cos(\omega t))\hat{i} + (C\omega\sin(\omega t))\hat{j}$$

$$\begin{aligned}\text{Speed} &= \sqrt{(R\omega - C\omega\cos(\omega t))^2 + (C\omega\sin(\omega t))^2} \\ &= \sqrt{R^2\omega^2 - 2RC\omega^2\cos(\omega t) + C^2\omega^2\cos^2(\omega t) + C^2\omega^2\sin^2(\omega t)} \\ &= \sqrt{R^2\omega^2 - 2RC\omega^2\cos(\omega t) + C^2\omega^2}\end{aligned}$$

$$\text{Speed} = \omega \sqrt{R^2 - 2RC\cos(\omega t) + C^2}$$

$$\boxed{\text{Speed} = \omega(R+C)}$$

$$@ t = \frac{\pi}{\omega} \therefore \omega \sqrt{R^2 - 2RC\cos(\pi) + C^2}$$

$$= \omega \sqrt{R^2 + 2RC + C^2} = \omega \sqrt{(R+C)^2} = \omega(R+C)$$

c) Circular Path: UCM

$$\vec{a}(t) = C\omega^2\sin(\omega t)\hat{i} + C\omega^2\cos(\omega t)\hat{j}$$

$$a = \frac{v^2}{R_c}, R_c = \text{Radius}_c$$

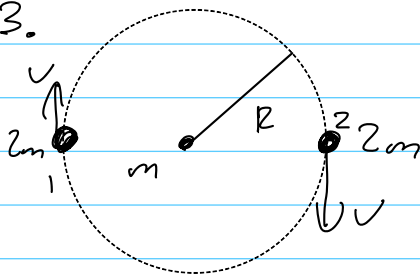
$$\therefore R_c = \frac{v^2}{a}$$

$$R_c = \frac{(\omega(R+C))^2}{C\omega^2}$$

$$\boxed{R_c = \frac{(R+C)^2}{C}}$$

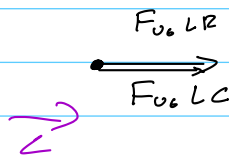
$$\begin{aligned}|\vec{a}(t)| &= \sqrt{(C\omega^2\sin(\omega t))^2 + (C\omega^2\cos(\omega t))^2} \\ &= \sqrt{C^2\omega^4(\sin^2(\omega t) + \cos^2(\omega t))} \\ &= C\omega^2\end{aligned}$$

3.

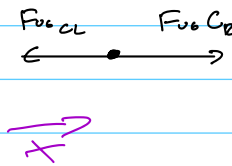


Given  
 $M_{\text{outer}} = 2m$  for both  
 $\text{Radius} = R$   
 $\text{Small Star Center } M_{\text{center}} = m$

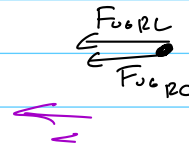
a) FBD Outer L



FBD Center



FBD Outer R



b) By N3L  $F_{OLC} = F_{CLL}$ . By symmetry we can see that the right outer star as well. The forces balance each other out results in zero net force on center.

c) Net Force is Vector Sum

$$\begin{aligned} F_{\text{Net}} &= F_{LR} + F_{LC} \\ &= \frac{G M_L m_R}{(2R)^2} + \frac{G M_L m_C}{(R)^2} \\ &= \frac{G(2m)(2m)}{4R^2} + \frac{G(2m)(m)}{R^2} \\ &= \frac{6m^2}{R^2} + \frac{2m^2}{R^2} = \frac{8m^2}{R^2} \end{aligned}$$

Vector Sum Magnitude =  $\boxed{\frac{8 \cdot G \cdot m^2}{R^2}}$

d) CGM Circular Path

$$a_c = \frac{v^2}{R}$$

$$F_{LC} = m_L a_c$$

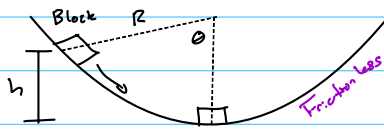
$$\frac{3 \cdot G \cdot m^2}{R^2} = m_L \left( \frac{v^2}{R} \right)$$

$$\frac{3 \cdot G \cdot m^2}{R^2} = 2m \frac{v^2}{R}$$

$$v^2 = \frac{3 \cdot G \cdot m}{2R} \quad v = \sqrt{\frac{3 \cdot G \cdot m}{2R}}$$

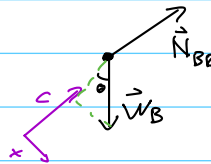
## Practice Exam 2

1.



- $h = R(1 - \cos \theta)$ ,  $R$  and  $\theta$  given
- Block of Mass  $m$  @ rest

a) FBD Block



b) NZL Block c-dr

$$F_c = m_B a_{\text{c}} \quad \text{UCM} \quad a_c = \frac{v^2}{R}$$

$$N_{Bc} - W \cos \theta = 0 \quad \text{Released @ rest} \Rightarrow v = 0$$

$$\boxed{N_{Bc} = mg \cos \theta} \quad \therefore a_c = \frac{0}{R} = 0$$

c) NZL Block x-dr

$$F_x = m_B a_{Bx}$$

$$W \sin \theta = m_B a_x$$

$$\boxed{a_x = g \sin \theta}$$

d)  $W_{\text{cons}} = -\Delta U_{\text{cons}}$

$$U_i = mgy, \quad W_{\text{cons}} = -(mgy - mgy')$$

$$= mgy - mgy'$$

Let  $y' = 0$  and  $y = h = R(1 - \cos \theta)$

$$\boxed{W_{\text{cons}} = mgR(1 - \cos \theta)}$$

e) Con ME

Conditions: No Friction, Normal Force does no work, and doesn't is conservative so we can use it!

e) Block = After

$$E_{\text{Tot}} = E'_{\text{Tot}}$$

$$U + K = U' + K'$$

$$mgy + \frac{1}{2}mv^2 = mgy' + \frac{1}{2}mv'^2$$

$$y' = 0, \quad y = R(1 - \cos \theta), \quad v = 0$$

$$mg(R(1 - \cos \theta)) = \frac{1}{2}mv'^2$$

$$v'^2 = 2gR(1 - \cos \theta)$$

FBD Block Bottom



NZL Block Bottom c-dr

$$F_c = m_B a_{Bc} \quad \text{UCM} \quad a_c = \frac{v^2}{R}$$

$$N - W = m \left( \frac{v^2}{R} \right)$$

$$N = m \left( \frac{2gR(1 - \cos \theta)}{R} \right) + mg$$

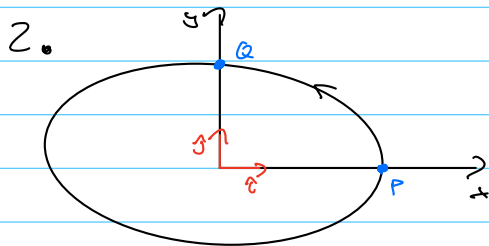
$$N = mg(2(1 - \cos \theta) + 1)$$

$$N = mg(2 - 2\cos \theta + 1)$$

$$N = mg(3 - 2\cos \theta)$$

$$N_{Bc} = mg(3 - 2\cos \theta) \quad \text{so} \quad N_{cB} = mg(3 - 2\cos \theta) \quad \text{by N32}$$

$$\boxed{N_{cB} = mg(3 - 2\cos \theta) \text{ points downwards}}$$



Given

- Particle mass  $m$
- Oval Path
- $\vec{r}(t) = 2A \cos(\omega t) \hat{e}_x + A \sin(\omega t) \hat{e}_y$   
 $\hookrightarrow A$  and  $\omega$  given and positive

a) Position Vector & Components & coordinates

$$\vec{r}(0) = 2A \cos(0) \hat{e}_x + A \sin(0) \hat{e}_y$$

$$\vec{r}(0) = 2A \hat{e}_x + 0 \hat{e}_y$$

$$t=0: (2A, 0)$$

$$\text{Speed} = A\omega$$

$$\vec{v}(t) = \frac{d}{dt} \vec{r}(t)$$

$$\vec{v}(t) = -2A\omega \sin(\omega t) \hat{e}_x + A\omega \cos(\omega t) \hat{e}_y$$

$$\vec{v}(0) = -2A\omega \sin(0) \hat{e}_x + A\omega \cos(0) \hat{e}_y$$

$$= A\omega$$

b) NZL  $\vec{F}_{\text{net}} = m \vec{a}$

$$\vec{a}(t) = -2A\omega^2 \cos(\omega t) \hat{e}_x - A\omega^2 \sin(\omega t) \hat{e}_y$$

$$\vec{a}(0) = -2A\omega^2 \cos(0) \hat{e}_x - A\omega^2 \sin(0) \hat{e}_y$$

$$= -2A\omega^2 \hat{e}_x$$

$$\vec{F}_{\text{net}} = -2A\omega^2 m \hat{e}_x$$

c)  $t = \frac{\pi}{2\omega}: \vec{a}(\frac{\pi}{2\omega}) = -2A\omega^2 \cos(\frac{\pi}{2}) \hat{e}_x - A\omega^2 \sin(\frac{\pi}{2}) \hat{e}_y$   
 $= 0 \hat{e}_x - A\omega^2 \hat{e}_y$

$$\vec{v}(\frac{\pi}{2\omega}) = -2A\omega \sin(\frac{\pi}{2}) \hat{e}_x + A\omega \cos(\frac{\pi}{2}) \hat{e}_y$$

$$= -2A\omega \hat{e}_x + 0 \hat{e}_y$$

$$R_c = \frac{(2A\omega)^2}{A\omega^2} = \frac{4A^2\omega^2}{A\omega^2} = 4A$$

$$a_c = \frac{v^2}{R_c} \quad \text{UCM Assumed}$$

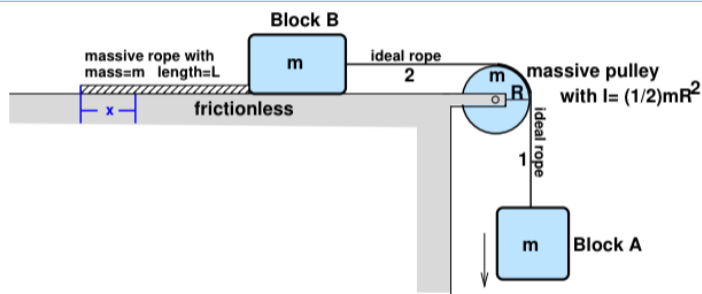
$$R_c = \frac{v^2}{a_c}$$

$$v = \sqrt{(2A\omega)^2} = 2A\omega$$

$$a_c = \sqrt{(-A\omega^2)^2} = A\omega^2$$

$$\therefore R_c = 4A$$

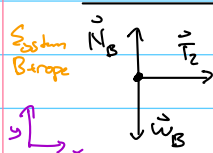
3.



Given

- A and B mass  $m$
- Ideal Rope and Massive
- Pulley: mass  $m$ , Radius  $R$ ,  $I = \frac{1}{2}mR^2$
- B on frictionless
- Massive Rope mass  $m$  length  $L$

a) FBD B



NZL B x-dir

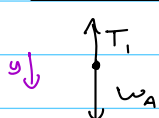
$$F_{Bx} = m_B a_{Bx}$$

$$T_2 = 2m a_B$$

Kinematic Constraint

$$a_B = a_A \quad \text{move together}$$

FBD A



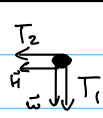
NZL A y-dir

$$F_{Ay} = m_A a_{Ay}$$

$$W_A - T_1 = m_A a_A$$

$$mg - T_1 = m a_B$$

FBD Pulley



NZL Rot Pulley

$$\tau_{\text{net}} = I \alpha$$

$$\tau_H + \tau_W + \tau_{T1} + \tau_{T2} = \frac{1}{2}mR^2 \alpha$$

$$0 + 0 + R T_1 \sin(90^\circ) + R T_2 \sin(90^\circ) = \frac{1}{2}mR^2 \alpha$$

$$T_1 - T_2 = \frac{1}{2}mR \alpha$$

$$\text{R.O.C. } a = R \alpha \quad \left. \begin{array}{l} T_1 - T_2 = \frac{1}{2}mR \left(\frac{a}{R}\right) \\ T_1 - T_2 = \frac{1}{2}m a_B \end{array} \right\}$$

$$\alpha = \frac{a_B}{R} \quad T_1 - T_2 = \frac{1}{2}m a_B$$

3 equations:  $T_2 = 2m a_B$   
 $T_1 = m(g - a_B)$   
 $T_1 - T_2 = \frac{1}{2}m a_B$

$$mg - m a_B - 2m a_B = \frac{1}{2}m a_B$$

$$mg - 3m a_B = \frac{1}{2}m a_B$$

$$mg = \frac{7}{2}m a_B$$

$$a_B = \frac{2}{7}g$$

$$T_2 = 2m a_B$$

$$T_2 = 2m \left(\frac{2}{7}g\right)$$

$$T_2 = \frac{4}{7}mg$$

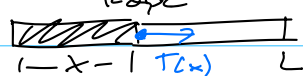
$$T_1 = m(g - a_B)$$

$$T_1 = m\left(g - \frac{2}{7}g\right)$$

$$T_1 = mg\left(1 - \frac{2}{7}\right)$$

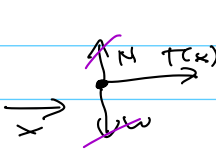
$$T_1 = \frac{5}{7}mg$$

b) System Rope



$$m_{\text{rope}} = \left(\frac{x}{L}\right)m$$

FBD System



NZL System x-dir

$$F_{\text{sys}x} = m_{\text{sys}} a_{\text{sys}}$$

$$a_{\text{sys}} = \frac{2}{7}g \quad \text{from (a)}$$

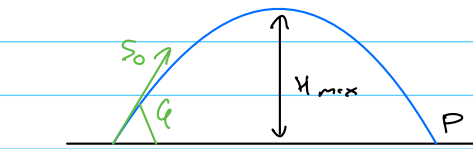
$$m_{\text{sys}} = \left(\frac{x}{L}\right)m$$

$$T(x) = \left(\frac{x}{L}\right)m \left(\frac{2}{7}\right)g$$

$$\therefore T(x) = \frac{2mgx}{7L}$$

### Practice Exam 3

1.



$S_0$  given,  $\phi$  given. Const. Speed

b)  $a_c = \frac{v^2}{R_c}$  Centripetal component of acceleration

a)  $\vec{r}(t) = (x_0 + v_{x0}t)\hat{i} + (y_0 + v_{y0}t - \frac{1}{2}gt^2)\hat{j}$

$x_0 = 0, v_{x0} = S_0 \cos \phi$

$y_0 = 0, v_{y0} = S_0 \sin \phi$

$\vec{v}(t) = v_{x0}\hat{i} + (v_{y0} - gt)\hat{j}$

$\vec{v}(t) = S_0 \cos \phi \hat{i} + (S_0 \sin \phi - gt)\hat{j}$

$v_x = S_0 \cos \phi, v_y = 0 @ H_{max}$

$v_y = S_0 \sin \phi - gt, 0 = S_0 \sin \phi - gt \rightarrow t = \frac{S_0 \sin \phi}{g}$

$y = y_0 + v_{y0}t - \frac{1}{2}gt^2, y_0 = 0, v_{y0} = S_0 \sin \phi, t = \frac{S_0 \sin \phi}{g}$   
 $H = S_0 \sin \phi \left( \frac{S_0 \sin \phi}{g} \right) - \frac{1}{2}g \left( \frac{S_0 \sin \phi}{g} \right)^2$

$H = \frac{(S_0 \sin \phi)^2}{g} - \frac{(S_0 \sin \phi)^2}{2g} \rightarrow H = \frac{(S_0 \sin \phi)^2}{2g}$