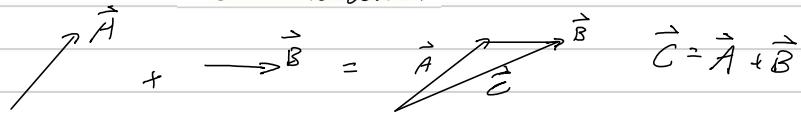


Vectors and 2D kinematics

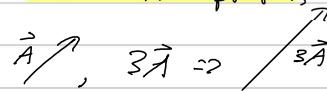
Recall: Vectors have both magnitude and direction.

Vector Addition

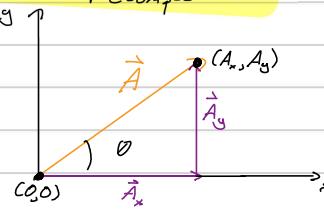


Length = "magnitude" $|\vec{A}|$

Scalar Multiplication



Vector Decomposition



Vectors are defined by magnitude!

$|\vec{A}| = \text{magnitude} = A_{\text{mag}} = \sqrt{A_x^2 + A_y^2}$
Direction = θ (using reference angle)

Or we can break it into components!
 $\vec{A} = \vec{A}_x + \vec{A}_y$

To find A : f A_x, A_y given: $A = \sqrt{A_x^2 + A_y^2}$ using
To find θ : f A_x, A_y given: $\theta = \arctan\left(\frac{A_y}{A_x}\right)$ Cross

Cartesian Basis Unit Vector



Don't use $< >$
Notation is misleading!

Unit Vector = length of one!



Means...

$$\begin{aligned}\vec{A}_x &= A_x \hat{i} \\ \vec{A}_y &= A_y \hat{j}\end{aligned}$$

$$\therefore \vec{A} = A_x \hat{i} + A_y \hat{j}$$

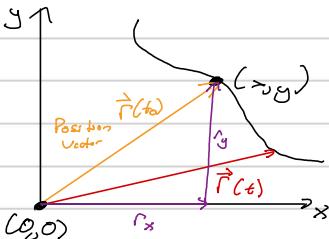
Vector Representation

Example: $\vec{C} = \vec{A} + \vec{B}$

$$\begin{cases} \vec{A} = A_x \hat{i} + A_y \hat{j} \\ \vec{B} = B_x \hat{i} + B_y \hat{j} \end{cases} \therefore \vec{C} = (A_x + B_x) \hat{i} + (A_y + B_y) \hat{j}$$

Path Integrals will Come Later

Kinematics w/ Vectors



$$\vec{r} = \vec{r}_x + \vec{r}_y = r_x \hat{i} + r_y \hat{j}$$

Position Vectors = \vec{r} vector

whose components are its own coordinates

$$\vec{v} = \frac{dr_x}{dt} \hat{i} + \frac{dr_y}{dt} \hat{j}$$

$$\vec{a} = v_x \hat{i} + v_y \hat{j}$$

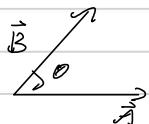
$$\vec{r} = \frac{d\vec{r}}{dt} \Rightarrow \vec{a} = \frac{d\vec{v}}{dt}$$

$$\begin{aligned}\vec{a} &= \frac{dv_x}{dt} \hat{i} + \frac{dv_y}{dt} \hat{j} \\ \vec{a} &= a_x \hat{i} + a_y \hat{j}\end{aligned}$$

Example: $\vec{r}(t) = (Bt^2) \hat{i} + (Ce^{-\frac{t}{\tau}}) \hat{j}$ t, B, C Given Constants Find Velocity and accel

$$\vec{v}(t) = (2Bt) \hat{i} - \left(\frac{C}{\tau} e^{-\frac{t}{\tau}}\right) \hat{j} \quad \text{and} \quad \vec{a}(t) = (2B) \hat{i} - \left(\frac{C}{\tau^2} e^{-\frac{t}{\tau}}\right) \hat{j}$$

Dot Products!



$$C = \vec{A} \cdot \vec{B} = AB \cos \theta$$

$$\hat{i} \cdot \hat{i} = 1, \hat{i} \cdot \hat{j} = 0$$

Using Decomposition

$$\therefore C = (A_x \hat{i} + A_y \hat{j}) \cdot (B_x \hat{i} + B_y \hat{j})$$

$$\text{FOLV} = A_x B_x (1) + A_x B_y (0) + A_y B_x (0) + A_y B_y (1)$$

$$C = A_x B_x + A_y B_y$$

Example: Related to HW4

Define: $\vec{r}(t) = R \cos(\omega t) \hat{i} + R \sin(\omega t) \hat{j}$, R, ω given

$$|\vec{r}| = \sqrt{x^2 + y^2} \rightarrow |\vec{r}| = \sqrt{(R \cos(\omega t))^2 + (R \sin(\omega t))^2}$$

Distance Magntude

Suppose



$$= \sqrt{R^2 (\cos^2(\omega t) + \sin^2(\omega t))} = \sqrt{R^2} = R$$

$$\text{Speed} = |\vec{v}| = \sqrt{v_x^2 + v_y^2}$$

$$\vec{v} = \frac{d\vec{r}}{dt} = -\omega R \sin(\omega t) \hat{i} + \omega R \cos(\omega t) \hat{j}$$

$$\therefore |\vec{v}| = \sqrt{R^2 \omega^2 (\sin^2(\omega t) + \cos^2(\omega t))} = R\omega$$

Omg! its the radial constant

Torce Derivatives of Functions of time!

$$\vec{a} = -R\omega^2 \cos(\omega t) \hat{i} - R\omega^2 \sin(\omega t) \hat{j}$$

$$|\vec{a}| = \sqrt{R^2 \omega^4 (\cos^2(\omega t) + \sin^2(\omega t))}$$

$$= \sqrt{R^2 \omega^4} = R\omega^2 = \frac{v^2}{R}$$

$$a = \frac{v^2}{R} = a_c$$

a_c is centripetal acceleration

Projectile Motion

x -direction:

No air resistance \rightarrow velocity is constant

$$a_x = 0$$

$$v_{x_0} = \text{Const} = v_{x_0}$$

$$x = x_0 + v_{x_0} t$$

y -direction:

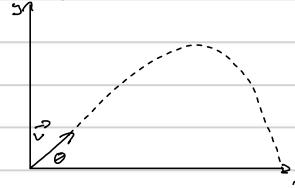
Free Fall

$$a_y = -g$$

$$V_y = V_{y_0} - gt$$

$$y = y_0 + V_{y_0} t - \frac{1}{2} g t^2$$

Graph



We see: $v_{x_0} = v_0 \cos \theta$ and $v_{y_0} = v_0 \sin \theta$

$$\therefore \vec{r} = x \hat{i} + y \hat{j}$$

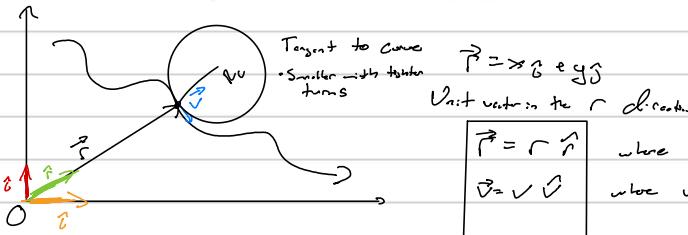
$$\text{In order to } \vec{r}(t) = (x_0 + v_{x_0} t) \hat{i} + (y_0 + v_{y_0} t - \frac{1}{2} g t^2) \hat{j}$$

$$\text{Position shown above} \quad \vec{r}(t) = (v_{x_0} t) \hat{i} + (v_{y_0} t - \frac{1}{2} g t^2) \hat{j}$$

$$\vec{a}(t) = 0 \hat{i} - g \hat{j}$$

Motion of Projectile in Vector Form

Example: A particle on a given path



$$\vec{r} = r \hat{r}$$

$$\vec{v} = v \hat{v}$$

where $r = |\vec{r}|$ and $|\hat{r}|=1$ r is mag, \hat{r} is dir

where $v = |\vec{v}|$ and $|\hat{v}|=1$ v is mag, \hat{v} is dir

$$\vec{a} = a_x \hat{i} + a_y \hat{j}$$

$$\vec{a} = a_r \hat{r} + a_\theta \hat{\theta}$$

Tangential Component Centripetal Component

R_c is radius of curvature

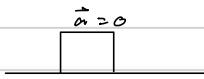
\hat{c} is \hat{r} to $\hat{\theta}$

$$\text{magnitude } a_r = \frac{dv}{dt} (\text{v}) \quad \text{vloc w/ respect to speed}$$

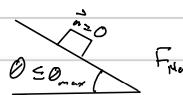
$$a_c = \frac{v^2}{R_c}$$

Both depend on speed

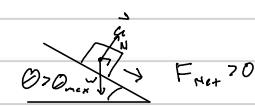
Forces at Angles



$$F_{\text{Net}} = 0$$



$$F_{\text{Net}} = 0$$



$\theta > \theta_{\max}$
 θ_{\max} is the maximum angle required for the body to remain at rest.

Steps

- (1) FBD
- (2) Coordinate System
- (3) Kinematic Constraint
- (4) NZL

(1.1) Decompose Forces

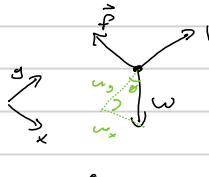
Decomposing Necessary forces into their necessary components is very important!

Suppose there is Friction



Given m, θ ; Find (i) all forces
 $a=0$ as others?

FBD Block



$$\begin{aligned} \text{N: } F_x &= ma_x & \text{y: } F_y &= mg \\ w_x - f &= 0 & N - w_y &= 0 \\ w_x = w \sin \theta &= mg \sin \theta & w_y = w \cos \theta &= mg \cos \theta \\ mg \sin \theta - f &= 0 & N &= mg \cos \theta \\ f &= mg \sin \theta & \end{aligned}$$

$$a_x = 0 \quad a_y = 0$$

NZL

$$\begin{aligned} w_x &= ma_x & \text{or } w_x = ma \\ w \sin \theta &= ma & \end{aligned}$$

$$mg \sin \theta = ma \Rightarrow a = g \sin \theta$$

Small $\theta \rightarrow$ Small a
 Large $\theta \rightarrow$ Large a

Suppose there is Kinetic Friction

Given w, μ_k, θ Find (i) a
 (ii) all forces

NZL

$$\begin{aligned} \text{y: } F_y &= mg \\ S &\text{ame as fr} \\ b &\text{ut to fr left} \\ N &= mg \cos \theta \\ N &= mg \cos \theta \end{aligned}$$

$$\begin{aligned} w_x - f &= ma_x \\ mg \sin \theta - f &= ma \\ mg \sin \theta - \mu_k mg \cos \theta &= ma \\ mg(\sin \theta - \mu_k \cos \theta) &= ma \end{aligned}$$

$$a = g(\sin \theta - \mu_k \cos \theta)$$

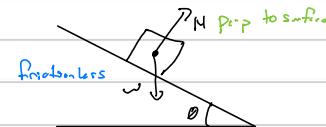
Given m, θ ; find (i) all forces

PBD Block

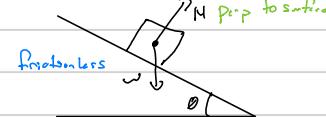
Kinematic Constraint

$$a_x = a$$

$$a_y = 0$$



Frictionless



$$F_{\text{Net}} = ma \quad x\text{-dir}$$

$$F_y = mg \quad \leftarrow \text{Normal Force and } y\text{-comp of the weight force}$$

$$N - mg = mg$$

$$N - mg \cos \theta = 0$$

$$N \sin \theta = 0$$

$$N = mg \cos \theta$$

$$\begin{aligned} \text{avoids having to break net force into components} \\ \text{in tipped coordinate} \\ w = w_x + w_y \\ w = w \sin \theta + w \cos \theta \\ \cos \theta = \frac{w_x}{w} \\ w_x = w \cos \theta \\ w = w \sin \theta \end{aligned}$$

$$F_{\text{Net}} = ma \quad x\text{-dir}$$

$$w_x = ma_x \quad \leftarrow \text{only } w_x \text{ in } x\text{-dir}$$

$$w \sin \theta = ma$$

$$mg \sin \theta = ma \Rightarrow a = g \sin \theta$$

Suppose there is Kinetic Friction

Given w, μ_k, θ Find (i) a
 (ii) all forces

NZL

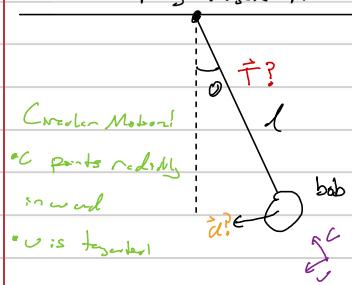
$$\begin{aligned} \text{y: } F_y &= mg \\ S &\text{ame as fr} \\ b &\text{ut to fr left} \\ N &= mg \cos \theta \\ N &= mg \cos \theta \end{aligned}$$

$$\begin{aligned} w_x - f &= ma_x \\ mg \sin \theta - f &= ma \\ mg \sin \theta - \mu_k mg \cos \theta &= ma \\ mg(\sin \theta - \mu_k \cos \theta) &= ma \end{aligned}$$

$$a = g(\sin \theta - \mu_k \cos \theta)$$

Pendulums

Jumping right in with an example



Pendulum bob with mass m released at rest angle θ
 Q1 What is the tension on string?

Q2 What is the acceleration of bob immediately after its released?

Given mass m , θ , and length of string l

a_c and a_v must be considered

FBD Bob

N2L : c -coord



$$F_c = m a_c$$

$$F_c = m \left(\frac{v^2}{l}\right) = m \left(\frac{0^2}{l}\right) = 0$$

$$T - w \cos \theta = 0 \quad \text{①}$$

$$T - mg \cos \theta = 0 \rightarrow T = mg \cos \theta \quad \text{②}$$

N2L : v -coord

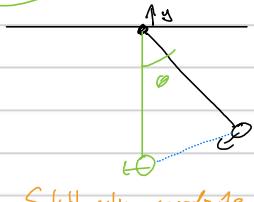
$$F_v = m a_v$$

$$w \sin \theta = m a_v$$

$$m g \sin \theta = m a_v$$

$$a_v = g \sin \theta$$

Q3 What is the Tension on string when bob at lowest position?



FBD Bob

N2L v -coord

$$F_c = m a_c$$

$$T - w = m \left(\frac{v^2}{l}\right)$$

$$T - mg = m \left(\frac{v^2}{l}\right) \text{ stop!}$$

v is unknown here!

FBD

Conservative

T does no work vs Δ top path
 $B_{\text{rel}} = A_{\text{rel}}$ ("B_{rel}" = release)

$E_{\text{tot}} > E_{\text{top}}$ ("A_{rel}" = lowest position)

$U + K = U' + K'$ Condition is met!

$$\Rightarrow T_{\text{mg}} = \frac{mv^2}{l} \text{ and } v^2 = \sqrt{2gl(1-\cos\theta)}$$

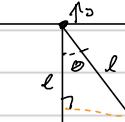
$$l + v^2 = \nu^2$$

$$T = mg + \frac{m(2gl(1-\cos\theta))}{l}$$

$$T = mg + 2mg(1-\cos\theta)$$

$$T = mg(1+2(1-\cos\theta))$$

$$T = mg(1+2-2\cos\theta) \rightarrow T = mg(3-2\cos\theta)$$



$$mg y + \frac{1}{2}mv^2 = mg y + \frac{1}{2}mv^2$$

$$y = -l \cos \theta, y = -l$$

$$V = 0, v^2 = ?$$

$$mg(-l \cos \theta) + \frac{1}{2}mv^2 = mg(-l) + \frac{1}{2}mv^2$$

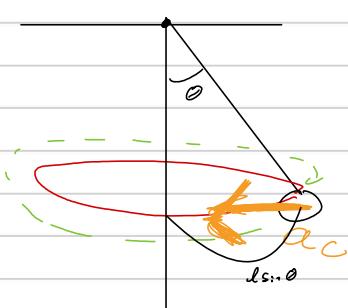
$$-mg l \cos \theta = -mg l + \frac{1}{2}mv^2$$

$$gl - gl \cos \theta = \frac{1}{2} v^2$$

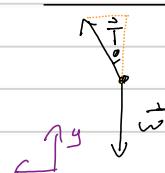
$$\sqrt{v^2} = \sqrt{2gl(1-\cos\theta)}$$

$$\therefore \sqrt{v^2} = \sqrt{2gl(1-\cos\theta)}$$

Central Pendulums



FBD Bob



N2L y -coord

$$F_y = m a_y, a_y = 0$$

$$T \cos \theta - w = 0$$

$$T = \frac{mg}{\cos \theta}$$

N2L x -coord

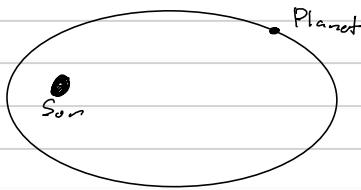
$$F_c = m a_c$$

$$T \sin \theta = m \frac{v^2}{l \sin \theta}$$

$$\frac{mg \sin \theta}{\cos \theta} = \frac{mv^2}{l \sin \theta}$$

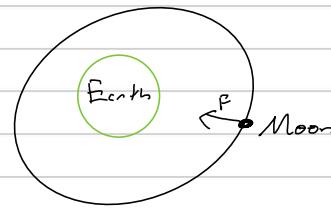
$$mg \tan \theta = \frac{mv^2}{l \sin \theta}, \text{ solve for } v$$

Planets!



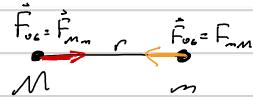
Kepler's 3rd Law (K3L)

$$\frac{P^2}{R^3} = \frac{(Period)^2}{("radius")^3} = \text{constant?}$$



Universal Gravity

$$F_{\text{Grav}} = \frac{G \cdot M \cdot m}{r^2}$$



G = Gravitational Constant

$$= 6.67 \times 10^{-11} \text{ N} \cdot \text{m}^2 \cdot \text{kg}^{-2}$$

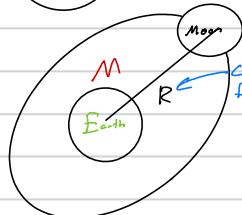
we only note it b/c of earth's size
it's ~~good~~ miss



$$\Rightarrow F_{\text{Grav}} = \frac{G M_E m}{R_E^2} = w = mg \rightarrow g_E = \frac{G M_E}{R_E^2}$$

$$F_{\text{Grav}} = \frac{G (1)(1)}{1} = G \quad \text{b/c anything out will}$$

where "g" comes from



Situation $M \gg m$
Orbit \approx Circular

NZL Moon

$$F_{\text{Moon}} = m a_c$$

$$\frac{G M_E m}{R^2} = m \frac{v^2}{R}$$

$$\text{Solve for } \frac{D^2}{R^3}$$

$$\rightarrow \frac{2\pi/R}{P} = v$$

displace
time

K3L back!

$$\int \frac{GM}{R^2} = \frac{4\pi^2 R^2}{P^2} \rightarrow GM = 4\pi^2 \left(\frac{R^3}{P^2} \right) \rightarrow \frac{P^2}{R^3} = \frac{4\pi^2}{GM}$$

Force Menu

Symbol	what	Known?	Conservative?	Contact?
w	weight	$w = mg$	Yes	No
N	Normal Push	No	No	Yes
T	Tension Pull	No	No	Yes
F	Friction	Sometimes	Absolutely No!	Yes ← known if kinetic friction unk
F_{Grav}	Universal Gravity	Yes	Yes	No
F_{sp}				

Hooke and Springs

"Spring"

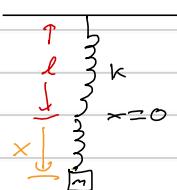
$$F_{\text{sp}} = k|x| \quad x \text{ is displacement from equilibrium length}$$

Compressed or stretched
 x is the change in length

$$F_{\text{sp}} = k|x|$$

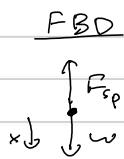
k = spring constant
 x = displacement relative to relaxed spring

Example



What is x s.t. mass m is in equilibrium? → $x = 0$

FBD



NZL x

$$F_x = m a_c \rightarrow 0$$

$$w = mg$$

$$w - F_{\text{sp}} = 0$$

$$F_{\text{sp}} =$$

$$mg - kx = 0$$

$$x = \frac{mg}{k}$$

You can only
assume $a_c = 0$
when mass
or object is in
equilibrium

Definition of Potential Energy

$$U_F(P) - U_{ref}(P_{ref}) = - \int_{P_{ref}}^P \vec{F} \cdot d\vec{r}$$

\vec{F} Force position
 P Path integral
 P_{ref} reference point
 $\vec{F} \cdot d\vec{r} = \text{Force} \theta$
 Change w/ time

$$\text{Work done by } \vec{F} = -\Delta U_F$$

Conservative force

Computing the integral

Weight \vec{w} and P_y

Pot. Energy

$$U_w(y) - U_{ref}(y_{ref}) = - \int_{y_{ref}}^y \vec{w} \cdot d\vec{r}$$

3 for weight

$$U_w(y) - U_{ref}(y_{ref}) = - \int_{y_{ref}}^y \vec{w} \cdot d\vec{r} = mg dy \cos(180^\circ)$$

$$U_w(y) - U_{ref}(y_{ref}=0) = - \int_{y=0}^y (-mg dy) = mg dy$$

Set up P to 0 in

Physics, not always

true in PHYS122

$$U_w(y) = mgy$$

Just an easy integral

Calculating for Spring Force

$$U_{sp}(x) - U_{ref}(x_{ref}) = - \int_{x_{ref}}^x \vec{F}_{sp} \cdot d\vec{r}$$

$$U_{sp}(x) - U_{ref}(x=0) = - \int_{x=0}^{x_{ref}} (-K_x dx)$$

$$U_{sp}(x) = \frac{1}{2} K_x x^2$$

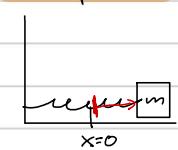
$$\vec{F}_{sp} = K_x \frac{d\vec{r}}{dx}$$

$$\vec{F}_{sp} \cdot d\vec{r} = K_x dx \cos(180^\circ)$$

$$|\vec{F}_{sp}| = -K_x dx$$

Spring force arguments

Example



Release at rest

at $x = x_i$

Find max speed

Given

K, m, x_i

Consider CoMFE when asked to find speed

"Before" = "After"

$$U_i + K = U_f + K'$$

$$\frac{1}{2} K x_i^2 + \frac{1}{2} m v_i^2 = \frac{1}{2} K x_f^2 + \frac{1}{2} m v_f^2$$

$$x = x_f, v = 0, x_i = 0 \quad v_f = v_p?$$

$$\frac{1}{2} K x_i^2 + 0 = 0 + \frac{1}{2} m v_p^2$$

$$\therefore v_p = x_i \sqrt{\frac{K}{m}}$$

Computing for Universal Gravity

$$U_{G6}(r) - U_{ref}(r_{ref}) = - \int_{r_{ref}}^r \vec{F}_{G6} \cdot d\vec{r}$$

Free fall

$$\vec{F}_{G6} \cdot d\vec{r} = \frac{GM_m}{r^2} dr \cos(180^\circ)$$

For goes to 0
when $r = \infty$

$$U_{G6}(r) - U_{ref}(r_{ref} = \infty) = - \int_{r=\infty}^r \left(-\frac{GM_m}{r^2} \right) dr$$

$$U_{G6}(r) = -\frac{GM_m}{r}$$



Conservative Forces

Weight \vec{w}

Spring Force $\vec{F}_{sp} = \frac{1}{2} K x^2$

Universal Grav. $\vec{F}_{G6} = -\frac{GM_m}{r^2}$

CoMFE

$$E_{Tot} = E_{Tot}'$$

$$U + K = U' + K'$$

U' can be a sum of different potential energies

Escape Velocity

ignores Air resistance, high R sends an object to f atmospheric as $\frac{1}{r^2}$ decreases



Atm

CoMFE

$$U + K = U' + K'$$

$$-\frac{GM_m}{r} + \frac{1}{2} mv^2 = -\frac{GM_m}{r'} + \frac{1}{2} mv'^2$$

$$-\frac{GM_m}{r} + \frac{1}{2} mv^2_{esc} = 0 + 0$$

$$\text{Algebra} \Rightarrow v_{esc} = \sqrt{\frac{2GM}{r}}$$

$$\approx 1.2 \times 10^4 \text{ m/s}$$

or 7 miles/sec

Thrown with some velocity
 v_{esc} . Lat. radius = R. Masses
of object not needed. Plan mass m.
thrown tangential. If it lands
is $\theta = 90^\circ$ and $r = \infty$. End to escape
velocity

More on Systems

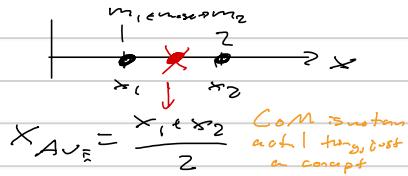
NCL for Systems

$$\vec{F}_{\text{net}} = M_{\text{sys}} \vec{a}_{\text{sys}}$$

Ext

works for systems of
n objects and directions
only with COM

Center of Mass (Com)



$$x_{\text{com}} = \frac{x_1 + x_2}{2}$$

COM is just a concept

Suppose $m_1 \neq m_2$, find weighted Avg

2 pts: $x_{\text{com}} = \frac{m_1 x_1 + m_2 x_2}{m_1 + m_2}$

weighted avg eqn

N pts: $x_{\text{com}} = \frac{m_1 x_1 + m_2 x_2 + \dots + m_n x_n}{m_1 + m_2 + \dots + m_n}$

$y_{\text{com}} = \frac{m_1 y_1 + m_2 y_2 + \dots + m_n y_n}{m_1 + m_2 + \dots + m_n}$

2 coordinates
with n bodies
with given masses
and positions

Time derivatives of Com

$$\frac{d x_{\text{cm}}}{dt} = v_{\text{cm}} = \frac{m_1 v_1 + m_2 v_2}{m_1 + m_2}$$

derivative of a sum is
the sum of derivatives

$$\frac{d v_{\text{cm}}}{dt} = a_{\text{cm}} = \frac{m_1 a_1 + m_2 a_2}{m_1 + m_2}$$

can be expanded to N points

NCL Systems

$$\vec{F}_{\text{net}} = M_{\text{sys}} \vec{a}_{\text{cm}}$$

vector be it
applies to N
number of bodies

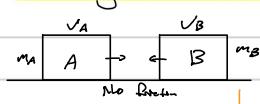
If $\vec{F}_{\text{net}} = 0 \Rightarrow$ "isolated" $\Rightarrow a_{\text{cm}} = 0$

$v_{\text{cm}} = \text{constant}$

$M_{\text{tot}} \cdot v_{\text{cm}} = P_{\text{tot}}$ COM Momentum!

Inertial reference frame is
defined to be at rest
moving with constant velocity

Total Elasto Collisions



$K = K'$ is w/ total elasto means

$$\therefore \frac{1}{2} m_A v_A^2 + \frac{1}{2} m_B v_B^2 = \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B v_B'^2$$

↳ $P_{\text{tot}} = P_{\text{tot}'}$

$m_A v_A + m_B v_B = m_A v_A' + m_B v_B'$

COM

COM

Analyze and a system of two

to solve for quadratically

use two method!

Bob Brown Method Eq

$$v_A - v_B = v_B' - v_A'$$

Only true in Total Elasto Collisions

Must be true!!!

Center of Mass Reference Frame

$$v_{\text{cm}} = \frac{m_A v_A + m_B v_B}{m_A + m_B} = \dots$$

can be calculated: P
 m_A, m_B, v_A , and v_B are
constants that move!

Relate to C.M Reference Frame $v_{\text{cm}} = 0$

Choose coordinate system
that moves with center of
mass (Com) of system!

$$v_{\text{cm}} = 0 \quad \text{if}$$

Example: $m_A = m_B = m$

$$v_A = v_0 \quad v_B = 0$$

$$(1) \quad m v_0 = m v_A' + m v_B'$$

$$(2) \quad v_0 = v_B' - v_A'$$

$$v_B' = v_0 + v_A'$$

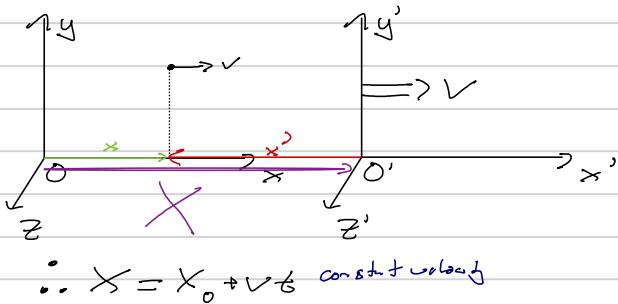
into (1)

$$m v_0 = m v_A' + m v_B'$$

$$2 v_0' = 0 \Rightarrow v_0' = 0$$

$$(2) \rightarrow v_B' = v_0$$

Reference Frames



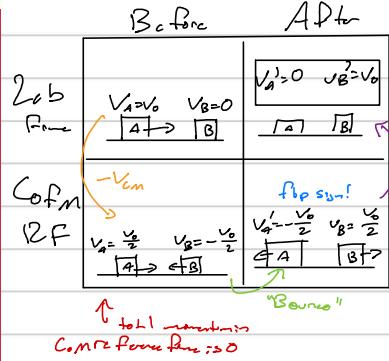
$$\begin{aligned} \bar{x} &= x - X \\ x' &= x - (x_0 + vt) \\ v' &= v - V \end{aligned}$$

$\bullet \Delta t = \text{stop time}$

\bullet Center-of-mass Ref. Frame

$$V_{\text{cm}} = V_{\text{lab}} - V_{\text{cm}}$$

Collisions 4-box method

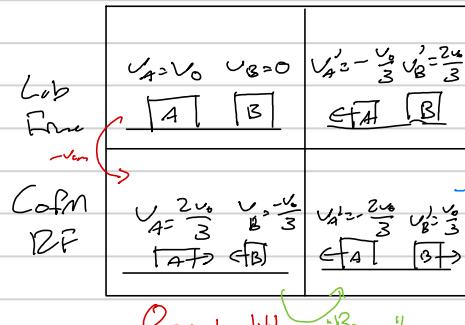


$$\begin{aligned} V_{\text{cm}} &= \frac{m_A V_A + m_B V_B}{m_A + m_B} \\ &= \frac{m_A V_0}{m_A + m_B} \quad m_A = m_B = m \\ V_{\text{cm}} &= \frac{m V_0}{2m} = \frac{V_0}{2} \\ V_A' &= 0 \\ V_B' &= V_0 \end{aligned}$$

Example: $m_A = m$, $m_B = 2m$

$$V_A = V_0, V_B = 0$$

Center-of-mass frame: $V_{\text{cm}} = \frac{m_A V_A + m_B V_B}{m_A + m_B} = \frac{m V_0 + 2m \cdot 0}{m + 2m} = \frac{V_0}{3}$



$$\begin{aligned} V_{\text{cm}} &= \frac{m_A V_0 + 0}{m_A + 2m} \\ &= \frac{m V_0}{3m} = \frac{V_0}{3} \\ V_A' &= -\frac{V_0}{3} \\ V_B' &= \frac{2V_0}{3} \end{aligned}$$

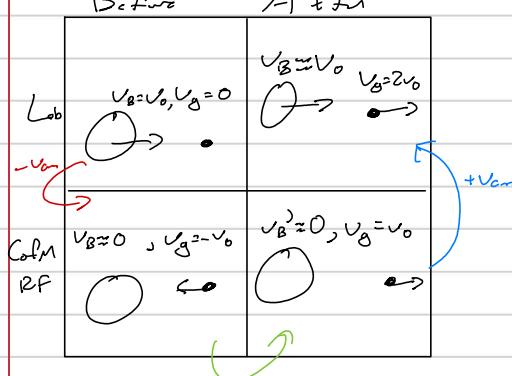
Example: Total elastic with rotating guns



$$\begin{aligned} V_{\text{cm}} &= \frac{m_B V_0 + 0}{m_B + m_G} \\ &\approx \frac{m_B V_0}{m_B + 0} = V_0 \end{aligned}$$

assuming light cart

If one mass much greater, assume it doesn't move

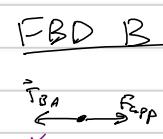
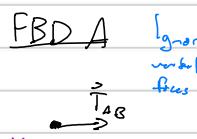


Solution principle



Frictionless

Final Tension



Example: Gun $m_A = m_B = m$ and F_{app} result of gun in 1D

Kinematic Constant

or C.C.L? $a_A = a_B = a$

Shortcut!

FBD System

$$\vec{P}_{\text{app}}$$

N2L x-dir System (A+B)

$$\begin{aligned} F_{x, \text{sys}} &= m_{\text{sys}} a_{\text{sys}} \\ F_{\text{app}} &= 2m a \end{aligned}$$

$$a = \frac{F_{\text{app}}}{2m}$$

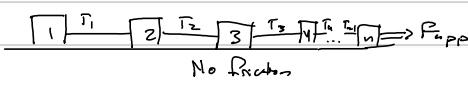
N2L A + dir

$$\begin{aligned} F_x &= m_A a_A \\ T_A &= m \left(\frac{F_{\text{app}}}{2m} \right) \end{aligned}$$

$$\therefore T = \frac{F_{\text{app}}}{2}$$

Calculated already

Example n number of blocks



Given: m_i , N , F_{app} . Find T_i

N2L new S_{distr}

$$F_{sys} = M_{sys} a_{sys}$$

$$T_i = (im) \frac{F_{app}}{N}$$

$$\therefore T_i = \frac{i}{N} F_{app}$$

$$a_1 = a_2 = a = a_{sys}?$$

PBD S_{distr} all blocks

To find T_i , do Rm a system

the boundary of the system is at the point

- for that force is applied

Suppose $i=3$

$$\rightarrow F_{app}$$

N2L \rightarrow dr S_{distr}

$$F_{sys} = M_{sys} a_{sys}$$

$$F_{app} = (Nm)a$$

$$a = \frac{F_{app}}{N}$$

N2L \rightarrow dr S_{distr}

PBD \rightarrow syst

New S_{distr} = leftmost i blocks

any friction PBD \rightarrow syst \rightarrow T_i

once cut! \Rightarrow

Example: Massive 120m

Massive Rope length l , F_{app} given

$$\dots \rightarrow F_{app}$$

Fraction loss

$T(x)$: Tension in rope x distance
from left end

FBD Rope

N2L 120m S_{distr}

$$F_{sys} = m_{sys} a$$

$$F_{app} = m a$$

$$a = \frac{F_{app}}{m}$$

New S_{distr} $T(x)$ \rightarrow app \rightarrow

left side up to x

FBD

N2L new S_{distr}

$$F_{sys} = m_{sys} a$$

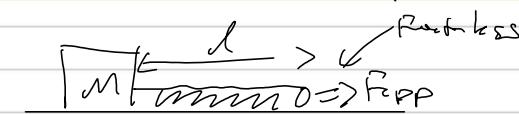
$$T(x) = (\frac{x}{120}) a$$

$$T(x) = (\frac{x}{120}) (\frac{F_{app}}{m})$$

ratio of rope
length times
tension

$$\therefore T(x) = (\frac{x}{120}) F_{app}$$

Example: Massive rope pulls block



Block m mass

$$F_{app}, l$$

Rope m mass

$$Br/1 T(x)$$

N2L w/ hole

$$F_{sys} = M_{sys} a$$

$$F_{app} = (Nm) a_{sys}$$

$$a_{sys} = \frac{F_{app}}{Nm}$$

New S_{distr}
= Boundary @ x

N2L new S_{distr}

$$F_{new} = M_{new} a$$

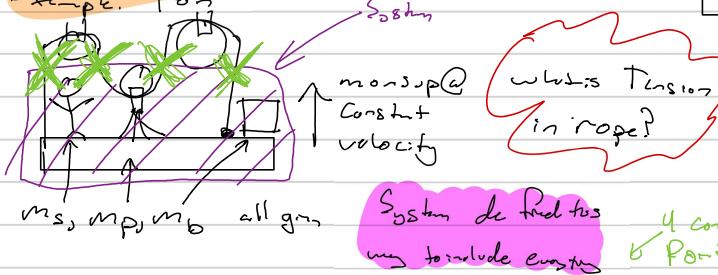
$$T(x) = ((\frac{x}{l}) m + M) \frac{F_{app}}{Nm}$$

simply

$$\therefore T(x) = \left(M + \frac{x}{l} m \right) \frac{F_{app}}{Nm}$$

$$T(x) = \left(\frac{Nm}{l} + M \right) \frac{F_{app}}{Nm}$$

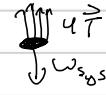
Example: F_{app}



System de Freitas
try to include const v

at constant
points

FBD S_{distr}



N2L S_{distr}

$$F_{sys} = m_{sys} a$$

$$4T - M_{sys} g = 0$$

$$T = \frac{M_{sys} g}{4}$$

$$\bar{T} = \frac{(m_s + m_p + m_b)g}{4}$$

Rotations!

Translational Motion

Kinematics: x, v, a

Dynamics: $F_{\text{net}} = ma$

N2L

Rotational Motion

Kinematics: θ, ω, α

Dynamics: $\tau_{\text{net}} = I\alpha$

N2L Rot

Review of P Motion

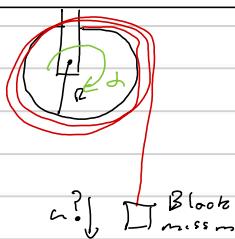
Components

New Force

\vec{H} = "Hinge Force"

axles, axes, pins

Example: Massive Pulley



Always assume string/rope is not slippery or stretchy

to Pulley

Muscle/Arms apply force $m_a g$ in a direction needed to satisfy N2L

Find accn of block

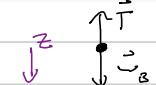
$a_x = ?$ $a_y = ?$

- Radius of Pulley
- Block of Mass m
- Pulley Mass is $4m$
 \rightarrow Pulley Mass = m_p
- $I_p = \frac{1}{2}m_p R^2$

\rightarrow guess for now!

Pulley has $\alpha = 0$ so
parts upward w/ constant

FBD Block



N2L Block

$$F_{Bx} = m_B a_{Bx}$$

$$W_B - T = m_B a$$

$$mg - T = m_B a$$

stop!

Two equations w/ two unknowns

$$\textcircled{1} \quad mg - T = m_B a$$

$$\textcircled{2} \quad H - 4mg - T = 0$$

Always $a = a_x$

Solve problem w/o
rotational motion

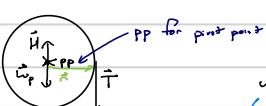
• Weight Force

always @ CDM

• Coordinate Systems

IS fixed w/ rotation

Extended FBD (xFBD) Pulley



← No longer a point-like object

N2L Rot Pulley

$$T_{\text{net}} = I\alpha$$

$$T_w + T_H + T_p = (\frac{1}{2}m_p R^2)\alpha$$

$$0 + 0 + RT \sin 90^\circ = \frac{1}{2}m_p R^2 \alpha$$

$$\therefore RT = \frac{1}{2}(4m)\alpha^2$$

Pulley $\alpha = \frac{\alpha}{R}$ Roll Clockwise

R.C is acceptable

$$RT = 2m R^2 \left(\frac{\alpha}{R}\right)$$

Recall: $mg - 2ma = ma$

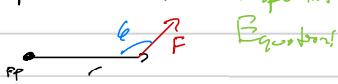
$$mg = 2ma + ma$$

$$g = a(2+1)$$

$$a = \frac{1}{3}g$$

Definition of Torque

$$\tau_p = r F \sin \theta$$

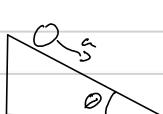


Rolling Contact

$a = 2r\alpha$ $v = wr$
Tangential acceleration
radial contact
normal contact

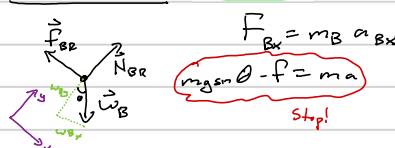
Very Helpful for HWG #6

Example: Ball on Ramp



- $\theta, R, m, I = \frac{2}{3}mR^2$
- Rolls w/ no slip or skid

FBD ball



N2L x-coord

$$F_{Bx} = m_B a_{Bx}$$

$$mg \sin \theta - f = ma$$

stop!

xFBD ball

N2L Rot ball

$$\tau_{\text{net}} = I\alpha$$

$$\tau_w + \tau_M + \tau_f = (\frac{2}{3}mR^2)\alpha$$

$$0 + R \cdot R \cdot (180^\circ) + Rf = \frac{2}{3}mR^2 \alpha$$

$$\alpha = \frac{a}{R}$$

$$Rf = \frac{2}{3}mR^2 \frac{a}{R}$$

$$f = \frac{2}{3}ma$$

$$mg \sin \theta - f = ma$$

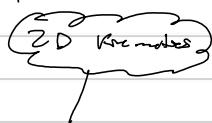
$$mg \sin \theta - \frac{2}{3}ma = ma$$

$$g \sin \theta = \frac{2}{3}a$$

$$a = \frac{3}{2}g \sin \theta$$

Review Lecture

Topics



Vectors

$$\vec{A} : |\vec{A}| \text{ is magnitude} = A$$

- Notation: $\vec{A} = A_x \hat{i} + A_y \hat{j}$

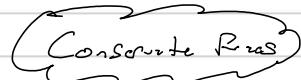
- Projectile Motion, v_{CM}

- Centripetal vs. Tangential Acceleration

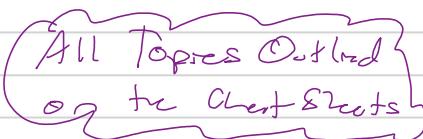
$$\hookrightarrow \vec{a} = \vec{a}_t + \vec{a}_c$$

$$a_v = \frac{d}{dt} (\text{speed})$$

$$a_c = \frac{v^2}{R_c} \text{ for } R_c \text{-radius of curvature}$$



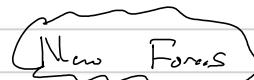
Know Formulas



Forces at AOs



- Tilted Coordinate System
in direction of Acceleration



Both Conservation

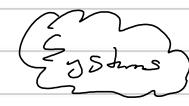
- Universal Grav. $F_{uc} = \frac{GMm}{r^2} \rightarrow \text{PWB's most favorite}$

- Sprng Forces $F_{sp} = k|x| \rightarrow \text{4-box Method}$
 $\hookrightarrow \text{displacement}$



- Moving Rope

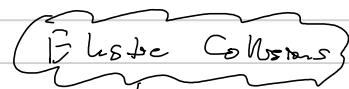
- Moving Pulley \leftarrow Rotational Motion



N2L

$$F_{Net} = M_{obj} a_{cm}$$

COM applied and calculated



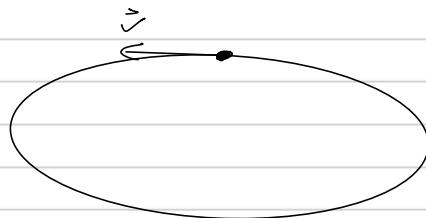
$$K = K_{tot}$$

PWB's most favorite

4-box Method

COM Frame vs. Lab Frame

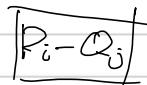
First Exam Topics
all Fair Game



$$\vec{v} = -v\hat{i}$$

Speed \rightarrow slows

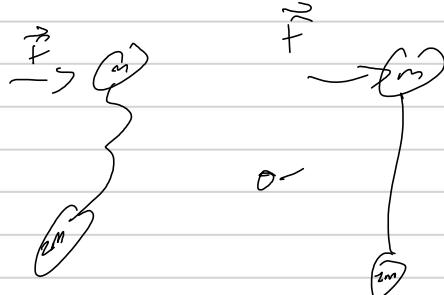
$$\frac{d(\text{speed})}{dt} = \text{negative}$$



$$\vec{a}_t = +P_i$$

$$\vec{a}_c = -Q_j$$

$$\text{Impulse} \equiv \int F dt \equiv \Delta p = F_{avg} \cdot \Delta t$$



More on Rotations

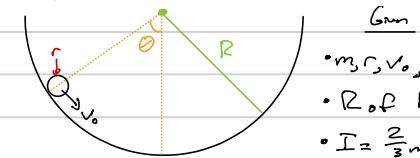
Summary of Important Collisions

$$\text{Torque: } \tau_p = r F_{\text{ext}} G$$

$$\text{NCL Rot: } \tau_{\text{net}} = I \alpha$$

$$\text{R.C.: } v = r\omega \text{ and } a = r\alpha$$

Example: Hollow ball rolls in bowl



- m, r, v_0, θ of ball, no slip/slide
- R of Bowl
- $I = \frac{2}{3}mr^2$

Values for Rotational Inertia (I)

Textbook lists my collist moment of inertia

$$I = C m R^2 \text{ for some const } C$$

Common Values for I:

$$I_{\text{loop}} = m R^2$$

$$I_{\text{disc}} = \frac{1}{2}m R^2$$

$$I_{\text{sphere}} = \frac{2}{5}m R^2$$

$$I_{\text{spine}} = \frac{2}{3}m R^2$$

$$I_{\text{rod CM}} = \frac{1}{12}ml^2$$

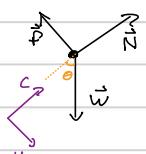
$$I_{\text{rod End}} = \frac{1}{3}ml^2$$

Good to have or
cheat sheet to reduce
error

Find Normal Force, Friction force, a

Assume $R \gg r$

FBD Ball



NCL c-coord centripetal

$$F_c = m a_c$$

$$N - m \cos \theta = m \frac{v^2}{R}$$

$$N = m(g \cos \theta + \frac{v^2}{R})$$

NCL v-coord tangential

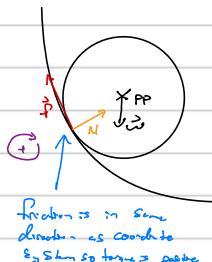
$$F_t = m a_t$$

$$w \sin \theta - f = m a_t$$

skip!

New Coord with Rotation!

\times FBD Bowl



NCL Rot

$$\tau_{\text{net}} = I \alpha$$

$$\tau_w + \tau_n + \tau_p = I \alpha$$

$$0 + 0 + r f \sin(90^\circ) = I \alpha$$

$$f = \frac{2}{3}mr^2 \alpha \quad \alpha = \frac{a_v}{r} \quad \text{R.C.}$$

$$f = \frac{2}{3}mr(\frac{a_v}{r})$$

skip!

Check if G is forced

force is possible for torque
if it is aligned with
coordinate system.

Two Equations, Two Unknowns

$$① m g \sin \theta - F = m a_v$$

$$② f = \frac{2}{3}m a_v$$

$$m g \sin \theta - \frac{2}{3}m a_v = m a_v$$

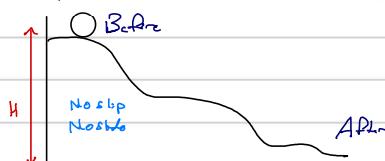
$$g \sin \theta = \frac{5}{3}a_v$$

$$a_v = \frac{3}{5}g \sin \theta$$

$$f = \frac{2}{3}m(\frac{3}{5}g \sin \theta)$$

$$f = \frac{2}{5}m g \sin \theta$$

Example: Ball rolling on slope



Given: Release @ rest

$$m, R, I = \frac{2}{5}mr^2 \text{ of ball}$$

H of ball initially

$$\text{Translational Kinetic: } K_T = \frac{1}{2}mv^2$$

$$\text{Rotational Kinetic: } K_R = \frac{1}{2}I\omega^2$$

$$\therefore K_{\text{Tot}} = K_T + K_R = \frac{1}{2}mv^2 + \frac{1}{2}I\omega^2$$

Friction does "no work" @ part of contact

$$E_{\text{tot}} = E'$$

$$U + K_T + K_R = U' + K'_T + K'_R$$

$$mgh + \frac{1}{2}mv^2 + \frac{1}{2}I\omega^2 = mgh' + \frac{1}{2}v'^2 + \frac{1}{2}I\omega'^2$$

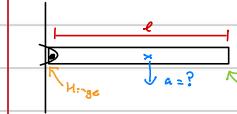
$$y = H, g = 0, v = 0, \omega = 0, \omega' = \frac{v}{R} \quad \text{R.C.}$$

$$mgh + \frac{1}{2}mv^2 + \frac{1}{2}(\frac{2}{5}mr^2)(\frac{v}{r})^2$$

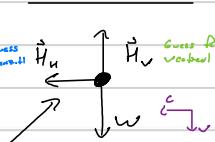
$$mgH = \frac{1}{2}mv^2 + \frac{1}{2}mv'^2 \quad v_f = v_d$$

$$mgH = \frac{3}{10}v^2 \rightarrow v_f = \sqrt{\frac{10}{3}gH}$$

Rod on Hinge



FBD Rod



Hinge Force is completely

unknown! Just assume

two arbitrary products

represent it. One is radial

one is horizontal

N2L Horizontal c-coord

$$F_c = m a_c$$

$$H_x = m \frac{v^2}{R} \quad @ \text{rest so } v=0$$

$$\therefore H_x = 0 \text{ works!}$$

N2L Vertical v-coord

$$F_v = m a_v$$

$$W - H_v = m a_v$$

$$mg - H_v = m a_v \quad \text{Stop!}$$

Final Acceleration of C.M.

@ release, Rod starts @ rest!

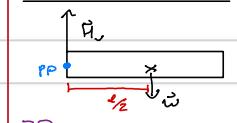
Not much speed, 1st start

the direction direction of H components tot add more on to get perp back.

C.M. is moving on circular paths so use v/c-coordinates.

PBD Rod

N2L Rot



$$\tau_{\text{Net}} = I \alpha \quad \text{known from Rod}$$

$$\tau_x + \tau_w = \frac{1}{3} ml^2 \alpha$$

$$O + \frac{l}{2} mg (\sin 45^\circ) = \frac{1}{3} ml^2 \alpha$$

$$\frac{1}{2} l mg = \frac{1}{3} ml^2 \left(\frac{\alpha}{r}\right)$$

$$\frac{1}{2} g = \frac{1}{3} 2 a_r$$

$$a_r = \frac{3}{4} g$$

R.C. a = r̄α

$$2 = \frac{a}{r}$$

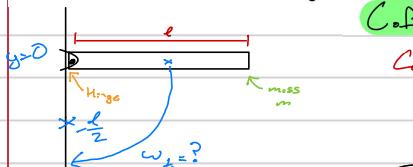
$$r = \frac{l}{2}$$

(?) Neglect H_x bc: +

is 0 from N2L

r is radial distn from
pp to C.M here!

Another Rod on Hinge



CoMFB

Conditions: wants conservation

Hinge does no work @ point of contact

contact.

Hinge Force does no work
bc it's @ point of contact

No force applied over a distn

Choose EndP Rods Reference frame! R!

Be Pre = After

$$E_{\text{tot}} = E'_{\text{tot}}$$

$$U + K_T + K_R = U' + K'_T + K'_R$$

$$U + O + K_{\text{Rad}} = U' + O + K'_{\text{Rad}}$$

$$mgy + O + \frac{1}{2} I w^2 = mgy' + O + \frac{1}{2} I w'^2$$

$$O + O + O = mg(-\frac{l}{2}) + \frac{1}{2} (\frac{1}{3} ml^2) w_p^2$$

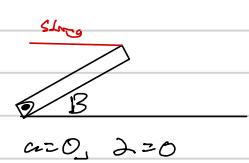
Algebra!

$$\frac{1}{2} mg l = \frac{1}{3} ml^2 w_p^2$$

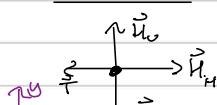
$$w_p = \sqrt{\frac{3g}{l}}$$

If you choose different
Reference Point I is diff!

Hinge with a string



FBD Rod



N2L y-dir

$$F_y = m a_y$$

$$H_x - W = 0$$

$$H_x = mg$$

N2L x-dir

$$F_x = m a_x$$

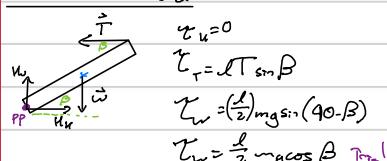
$$H_x - T = 0$$

$$T = mg \quad \text{Stop!}$$

At rest! So H_x and the

most hold each other

PBD Rod



$$\tau_u = 0$$

$$\tau_r = l T \sin \beta$$

$$\tau_w = \left(\frac{l}{2}\right) mg \sin(40 - \beta)$$

$$\tau_w = \frac{l}{2} mg \cos \beta \quad \text{Ans!}$$

Adolfi

Statics

Static Friction

$$f_s \leq \mu_s N$$

Constrained

Kinetic Friction

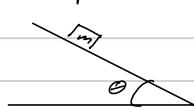
$$f_k = \mu_k N$$

Equal: f

For Slotes, $a=0$ and $\dot{x}=0$

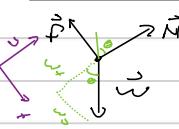
Use N2L to find N \rightarrow N2L to find $f_s \rightarrow$ Apply condition +

Example



Given: m , θ , μ_s
Find θ_{max}

FBD Block



N2L y-dir

$$F_y = m g \rightarrow 0$$

$$N - w \cos \theta = 0$$

$$N = w \cos \theta$$

N2L x-dir

$$F_x = m a_x \rightarrow 0$$

$$w \sin \theta - f_s = 0$$

$$f_s = w \sin \theta$$

Example shows $f_s \leq \mu_s N$

is not always! Good for
determining minimums!

App Condition 2: $f_s \leq \mu_s N$

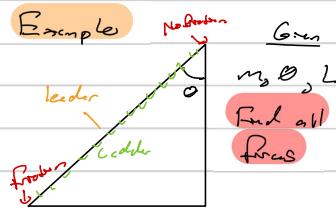
No in Equal Signs!

$$m g \sin \theta \leq \mu_s m g \cos \theta$$

$$\frac{\sin \theta}{\cos \theta} \leq \mu_s$$

$$\begin{aligned} &\Rightarrow \tan \theta \leq \mu_s \\ &\theta \leq \arctan \mu_s \\ &\therefore \theta_{max} = \arctan(\mu_s) \end{aligned}$$

Example

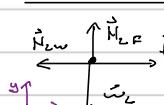


Given

$$m g, \theta, L$$

Find all
forces

FBD Ladder



N2L y-dir

$$F_y = m a_y \rightarrow 0$$

$$N_{WL} - w = 0$$

$$N_{WL} = w$$

$$w = m g$$

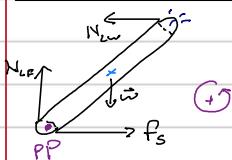
N2L x-dir

$$F_x = m a_x \rightarrow 0$$

$$f_s - N_{LW} = 0$$

$$f_s = N_{LW}$$

FBD Ladd.r



N2L Rot

$$\tau_{net} = I \alpha$$

$$\tau_{N_{LF}} + \tau_{f_s} + \tau_w + \tau_{N_{LW}} = 0$$

$$0 + 0 - \frac{1}{2} w \sin \theta + L N_{LW} \sin(90 - \theta) = 0$$

$$-\frac{1}{2} L m g \sin \theta + L N_{LW} \cos \theta = 0$$

Place PP w/ $w \sin \theta$ + kills
most forces

$$N_{LW} \cos \theta = \frac{1}{2} m g \sin \theta$$

$$N_{LW} = \frac{1}{2} m g \tan \theta$$

$$f_s - N_{LW} = 0$$

$$f_s = N_{LW}$$

$$f_s = \frac{1}{2} m g \tan \theta$$

Result: $\sin(90 - \theta) = \cos \theta$

$$\cos(90 - \theta) = \sin \theta$$