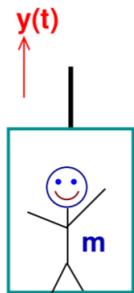


Practice Set

Problem 1)



Given

- Mass m of person
- $y(t) = At + Bt - Ct^2 + Dt^3$ for position
- Constants of $y(t)$: A, B, C, D
- $0 < t < T$ for interval where T is given

a) $v = \frac{dy}{dt} (y(t))$ definition of velocity

$$v(t) = \frac{d}{dt} (At + Bt - Ct^2 + Dt^3) = B - 2Ct + 3Dt^2$$

$$v(t=0) = B - 0 + 0 \quad v(0) = 0$$

b) $a = \frac{dv}{dt} (v(t))$ definition of acceleration

$$a(t) = \frac{d}{dt} (B - 2Ct + 3Dt^2) = -2C + 6Dt$$

$$v(t=T) = -2C + 6DT$$

c) N2L $F_{net} = ma$

$$\text{Net force} = 0, \text{ so } a = 0$$

$$0 = ma$$

$$a = 0$$

$$-2C + 6DT = 0$$

$$6DT = 2C$$

$$t = \frac{2C}{6D} \rightarrow t = \frac{C}{3D}$$

d) Effective weight

$$\vec{W}_{eff} = \vec{W}_{real} + \vec{W}_{fact}$$

$$= mg_{real} + mg_{fact}$$

$$g_{fact} = -A = 2C - 6Dt$$

$$\vec{W}_{eff} = -mg\hat{j} + m(2C - 6Dt)$$

$$= m(-g + 2C - 6Dt)$$

$$\vec{W}_{eff} = -m(g - 2C + 6Dt)$$

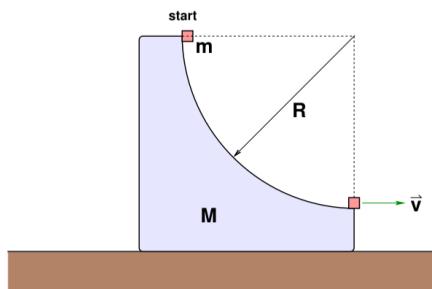
$$\text{At } t=0: W_{eff} = -m(g - 2C) \leftarrow -mg$$

$$\text{At } t=t: W_{eff} = -m(g - 2C + 6Dt) \downarrow \text{increasing}$$

$$\text{At } t=T: W_{eff} = -m(g - 2C + 6DT) \downarrow$$

At $t=0$, you feel lighter than normal, but $0 < t < T$ will cause you to feel slightly heavier, and post time T it follows that you will become increasingly heavy.

Problem 2)



Given

- Radius R of cube cut
- Frictionless path and surface
- Released at rest
- m of small block, M of cube

Let $v_A = \text{small block}, v_B = \text{big block}$

CofLM conditions: big and

Small block isolated, no
friction

Looking for v_A'

Before = After

$P_{\text{Tot}} = P'_{\text{Tot}}$

$$m_A v_A + m_B v_B = m_A v_A' + m_B v_B'$$

$$@ \text{rest} \quad 0 + 0 = m_A v_A' + m_B v_B'$$

$$v_B' = -\frac{m_A}{m_B} v_A' \quad \text{stop!}$$

CofME Conditions: frictionless

wants to conserve normal force

zero work \rightarrow to motion.

Before = After

$$U_{\text{Tot}} + K_{\text{Tot}} = U'_{\text{Tot}} + K'_{\text{Tot}} \quad \text{Not lone block does not have potential energy here}$$

$$m_A g y + \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B v_B'^2 = m_A g y + \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B v_B'^2$$

$$y = R, y = 0, v_A' = ?, v_B' = -\frac{m_A}{m_B} v_A' \\ m_A g R = \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B \left(-\frac{m_A}{m_B} v_A'\right)^2 \\ = \frac{1}{2} m_A v_A'^2 + \frac{1}{2} m_B \left(\frac{m_A}{m_B} v_A'\right)^2$$

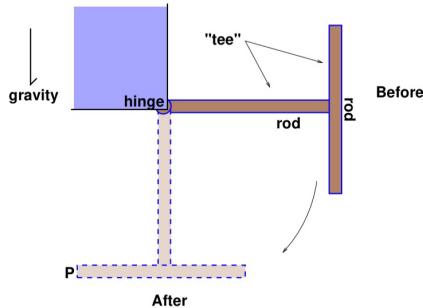
$$m_A g R = \frac{1}{2} m_A v_A'^2 + \frac{1}{2} \frac{m_A}{m_B} (m_A v_A'^2)$$

$$m_A g R = \frac{1}{2} m_A v_A'^2 \left(1 + \frac{m_A}{m_B}\right)$$

$$g R = \frac{1}{2} \left(1 + \frac{m_A}{m_B}\right) v_A'^2$$

$$v_A' = \sqrt{\frac{2 g R}{1 + \frac{m_A}{m_B}}}$$

Problem 3)



Given

- Rigid Body
- Each rod is length L mass m

a) Center of Mass Def

$$x=0 \quad \text{rod A} \quad x=\frac{L}{2} \quad \text{rod B}$$

$$m_{\text{rod A}} = m \quad m_{\text{rod B}} = m$$

$$x_{\text{cm}} = \frac{m_A x_A + m_B x_B}{m_A + m_B} = \frac{m \left(\frac{L}{2}\right) + m L}{2m} = \frac{\frac{3}{2}mL}{2m} = \frac{3L}{4}$$

b) Total Inertia $I_T = I_A + I_B$

I_A : Rod about end

$$I_A = \frac{1}{3}mL^2$$

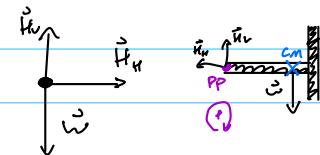
I_B : Rod about Cm, Parallel Axis Thm

$$I_B = I_{\text{cm}} + mD^2 = \frac{1}{12}mL^2 + mL^2$$

$$I_T = \frac{1}{3}mL^2 + \frac{13}{12}mL^2 = \frac{4}{12}mL^2 + \frac{13}{12}mL^2$$

$$I_T = \frac{17}{12}mL^2$$

c) FBD 'vec' \times FBD



N2L Rot $\tau_{\text{net}} = I\alpha$

$$\tau_h + \tau_w = \left(\frac{17}{12}mL^2\right)\alpha$$

Torque: $\tau = rF \sin\theta$

$$\rightarrow \tau_h = 0$$

$$\rightarrow \tau_w = \frac{3}{4}L(2mg)$$

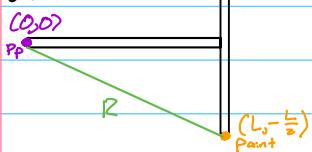
$$\frac{3}{2}mgL = \frac{17}{12}mL^2\alpha$$

$$\frac{3g}{22} = \frac{17}{12}\alpha$$

$$\alpha = \frac{12}{17}\cdot\frac{g}{L}$$

$$\alpha = \frac{18g}{17L}$$

d)



Total Two bonds α & ω

Point as a system with

mass $2m$.

CofME:

Conditions: $S_0 \neq \text{trans}$

obviously isolated,
forces constant, no work

$$B_{\text{CofME}} = A_{\text{CofME}}$$

$$E_{\text{Tot}} = E'$$

~~$$mg\cancel{\frac{1}{2}I_T \omega_T^2} = mg\cancel{\frac{1}{2}I_T \omega_T^2}$$~~

$$0 = 2mg\left(-\frac{3L}{4}\right) + \frac{1}{2}\left(\frac{17}{12}mL^2\right)\omega_T^2$$

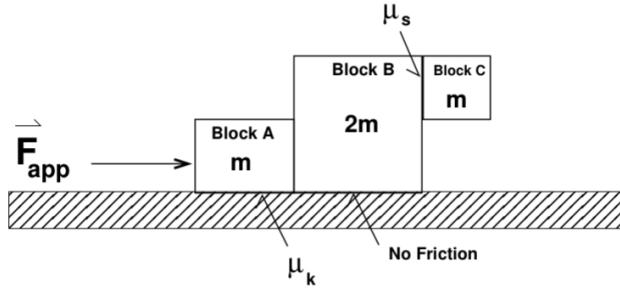
$$\frac{3gL}{2} = \frac{17}{24}L^2\omega_T^2$$

$$\frac{36g}{17} = L^2\omega_T^2$$

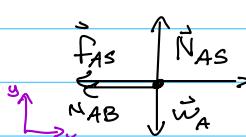
$$\omega_T^2 = \frac{36g}{17L^2}$$

$$\omega_T = \frac{36g}{17L}$$

Problem 4)



a) FBD A



b) Kinematic Friction

$$f_{AS} = \mu_k |N_{AS}|$$

NZL block A y-dir

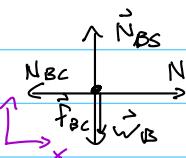
$$F_{\text{Net}} = m_A g \quad \text{No vertical!}$$

$$N_{AS} - w_A = 0 \quad N_{AS} = m_A g, \quad m_A = m \quad a_{sys} = a_x = ?$$

$$f_{AS} = \mu_k m_A g$$

Fricious leftward!

FBD B



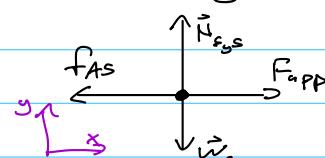
c) System, $a_A = a_B = a_C = a_x$

NZL System x-dir

$$F_{\text{Net}} = m_{sys} a_{sys}$$

$$m_{sys} = m + 2m + m = 4m$$

FBD System



Kinematic Constraint

$$F_{app} - f_{AS} = 4m a_x$$

$$\frac{F_{app} - \mu_k m g}{4m} = a_x$$

$$a_x = \frac{F_{app} - \mu_k m g}{4m}$$

d) NZL A x-dir

$$F_{\text{Net}} = m a_x$$

$$F_{app} - f_{AS} - N_{AB} = m \left(\frac{F_{app} - \mu_k m g}{4m} \right)$$

$$N_{AB} = F_{app} - \mu_k m g - \frac{1}{4}(F_{app} - \mu_k m g)$$

$$N_{AB} = \frac{3}{4}(F_{app} - \mu_k m g)$$

NZL $N_{AB} = N_{BA}$

$$N_{BA} = \frac{3}{4}(F_{app} - \mu_k m g)$$

Rightward!

e) NZL B x-dir

$$F_{\text{Net}} = m_B a_x$$

$$N_{BA} - N_{BC} = 2m a_x$$

$$\frac{3}{4}(F_{app} - \mu_k m g) - N_{BC} = 2m \left(\frac{F_{app} - \mu_k m g}{24m} \right)$$

$$\frac{3}{4}(F_{app} - \mu_k m g) - N_{BC} = \frac{1}{2}(F_{app} - \mu_k m g)$$

$$N_{BC} = \frac{1}{4}(F_{app} - \mu_k m g)$$

Leftward!

f) NZL C y-dir

$$F_{\text{Net}} = m g \quad \text{no net moment!}$$

$$f_{CB} - w_C = 0$$

$$f_{CB} = m g \quad \text{upward}$$

g) Static Friction Constraint

$$f_s \leq \mu_s N$$

NZL $N_{CB} = N_{BC}$

$$f_{CB} \leq \mu_s N_{CB}$$

$$= \frac{1}{4}(F_{app} - \mu_k m g)$$

$$m g \leq \mu_s \cdot \frac{1}{4}(F_{app} - \mu_k m g)$$

$$\frac{m g}{\mu_s} \leq F_{app} - \mu_k m g$$

$$F_{app} \geq \frac{\mu_m g}{\mu_s} + \mu_k m g$$

Given

F_{app} to block A

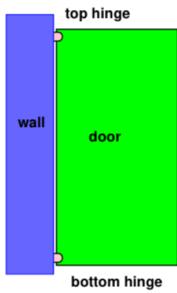
$A \text{ mass} = m, B \text{ mass} = 2m, C \text{ mass} = m$

C holds in place by static friction

B and C static friction μ_s

A and surface μ_k , B frictionless

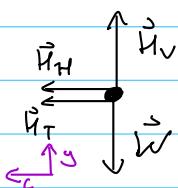
Problem 5)



Given

- mass m , height h , width w of door
- Bottom hinge applies no-tension force \vec{H}_B and H_B
- Top hinge $\parallel \parallel \parallel \vec{H}_T$, zero vertical force
- Door swings about hinge @ S not sped
- Door inertia about hinge: $I = \frac{1}{3}mw^2$

FBD Door



NZL Door y-dir

$$H_U - W = mg \quad \text{only horizontal}$$

$$H_U = W$$

$$H_U = mg$$

NZL c-dir

$$H_H + H_T = m\alpha_c \quad \alpha_c = \frac{\omega^2}{r}$$

$$H_H + H_T = m \frac{\Omega^2 r^2}{r} \quad R.C. \quad \omega = \Omega r$$

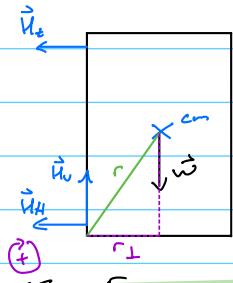
$$H_H + H_T = m \Omega^2 r \quad r = \frac{w}{2} \text{ from left}$$

$$H_H + H_T = m \Omega^2 \left(\frac{w}{2}\right)$$

Sub!

Door moves in path of $\frac{w}{2}$ circularly

x FBD Door



NZL Rot

$$\tau_{Net} = I \alpha \quad \text{constant speed}$$

$$\tau_{H_H} + \tau_{H_V} + \tau_{H_T} + \tau_{H_W} = 0$$

$$hH_H + r_\perp W = 0$$

$$hH_H + \frac{w}{2}mg = 0$$

$$H_H = \frac{wmg}{2h}$$

$$\frac{wmg}{2h} + H_H = m \Omega^2 \left(\frac{w}{2}\right)$$

$$H_H = m \Omega^2 \left(\frac{w}{2}\right) - \frac{wmg}{2h}$$

$$H_H = \frac{1}{2}mw \left(\Omega^2 - \frac{g}{h} \right)$$

$$H_H = \frac{1}{2}mw \left(\Omega^2 - \frac{g}{h} \right)$$

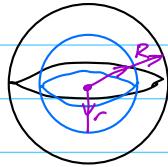
Problem 6)



Given

- Neglect air resistance
- Gravitational force on small body :- $\propto r^2$ (sphere is $\propto r^2$)
- M and R are mass and radius of earth
- Person has mass m (Assume earth uniform density)

$$a) F_{\text{ext}}(r) = \frac{G_M m}{r^2} \quad M \text{ is earth's mass}$$



$$\text{Volume of inner sphere: } \frac{4}{3}\pi r^3$$

$$\text{Volume of outer space: } \frac{4}{3}\pi R^3$$

Volume fraction of liner

$$V = \frac{V_1}{V_0} = \frac{\frac{4}{3}\pi r^3}{\frac{4}{3}\pi R^3} = \frac{r^3}{R^3}$$

Mass of her sphere

$$M = \frac{r^3}{R^3} \cdot M$$

$$F_{\text{ext}}(r) = \frac{Gm}{R^3} \cdot r$$

$$\therefore F_{\text{ext}}(r) = \frac{G(\frac{r^3}{R^3}M)m}{r^2} = \frac{GrM}{R^3}m$$

$$b) \text{ NZL } F_r = m a_r$$

$$\text{Attractive} \quad -F_{\text{ext}} = m \frac{d^2 r}{dt^2}$$

$$\frac{d^2 r}{dt^2} + F_{\text{ext}} = 0$$

$$\frac{d^2 r}{dt^2} + \frac{GM_m}{R^3} r = 0$$

$$\frac{d^2 r}{dt^2} + \frac{GM}{R^3} r = 0$$

c) 2nd order DE

$$\text{Characteristic polynomial: } s^2 + \frac{GMm}{R^3} = 0$$

$$s = \pm i \sqrt{\frac{GM}{R^3}}$$

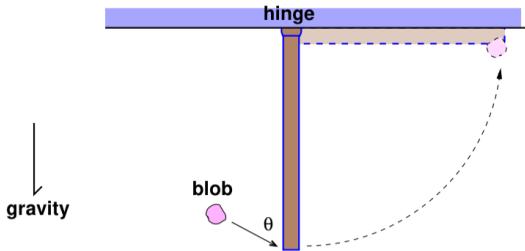
$$\text{Allow } \begin{cases} r(0) = R \\ r'(0) = v(0) = 0 \end{cases}$$

$$\begin{cases} \vec{r}(t) = K_1 \cos\left(\sqrt{\frac{GM}{R^3}} t\right) + K_2 \sin\left(\sqrt{\frac{GM}{R^3}} t\right) \\ \vec{r}'(t) = -\sqrt{\frac{GM}{R^3}} K_1 \sin\left(\sqrt{\frac{GM}{R^3}} t\right) + \sqrt{\frac{GM}{R^3}} K_2 \cos\left(\sqrt{\frac{GM}{R^3}} t\right) \end{cases}$$

$$\begin{cases} R = K_1 + 0 \\ 0 = 0 + \sqrt{\frac{GM}{R^3}} K_2 \quad K_2 = 0 \end{cases} \quad K_1 = R \rightarrow \vec{r}(t) = R \cos\left(\sqrt{\frac{GM}{R^3}} t\right)$$

d) According to Einstein's principle of equivalence you cannot perform any experiment to determine your location relative to the earth. Your reference frame accelerates at the same rate as gravity pulls you, so effective gravity is 0, which leads to a weightless feeling.

Problem 7)



Given

- Thin rod massless, length L
- Ideal hinge
- Blob stuck to rod @ θ to vert

CofME \rightarrow swing

Conditions: Ideal hinge does

No work, momentum conserved.

$$B_{\text{CofMe}} = A_{\text{Ptot}}$$

$$E_{\text{Tot}} = E'_{\text{Tot}} \quad \text{ends @ rotated } y=0$$

$$U_B + E_B + U_D + E_R = U'_B + E'_B + U'_R + E'_B$$

$$mgg + \frac{1}{2}mv_B^2 + mg y_B + \frac{1}{2}I\omega^2 = 0$$

$$-mgh + \frac{1}{2}mv_B^2 + mg\left(\frac{L}{2}\right) + \frac{1}{2}\left(\frac{1}{3}mL^2\right)\omega^2 = 0$$

$$-2mgL + mv_B^2 - mgh + \frac{1}{3}mL^2\omega^2 = 0$$

R.C. $v = \omega r \quad v_B = \omega L$ (start)

$$-3mgL + m(\omega L)^2 + \frac{1}{3}mL^2\omega^2 = 0$$

$$\omega^2\left(v_B L^2 + \frac{1}{3}L^2\right) = 3gL$$

$$\omega^2\left(\frac{4}{3}L\right) = 3g$$

$$\omega^2 = \frac{9}{4}\frac{g}{L} \rightarrow \omega = \frac{3}{2}\sqrt{\frac{g}{L}}$$

CofAM

Conditions: blob-rod system isolated

$$B_{\text{CofAM}} = A_{\text{Ptot}}$$

$$L_{\text{Tot}} = L'_{\text{Tot}}$$

$$L_B + L_R = L'_B + L'_R$$

$$mv_i L \sin \theta + 0 = mv_f L + I\omega \quad v = \omega r$$

$$\cancel{mv_i L \sin \theta} = \cancel{mv_f L} + \left(\frac{1}{3}mL^2\right)\omega$$

$$v_i \sin \theta = \omega L + \frac{1}{3}L\omega$$

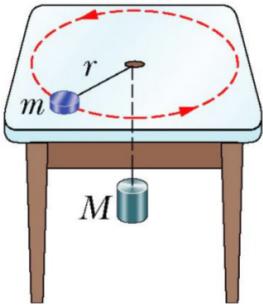
$$v_i \sin \theta = \frac{4}{3}L\omega$$

$$v_i = \frac{4L\omega}{3 \sin \theta}$$

$$v_i = \frac{2gL}{3 \sin \theta} \cdot \frac{3}{2} \sqrt{\frac{g}{L}} = \frac{2L}{\sin \theta} \sqrt{\frac{g}{L}}$$

$$v_i = \frac{2}{\sin \theta} \sqrt{gL}$$

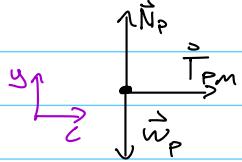
Problem 8)



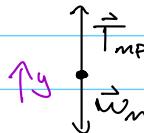
Given

- Small puck mass on frictionless surface
- Massless string holding unknown mass M
- Puck moves in c.c. with constant radius r , speed v

a) FBD Puck



FBD Mass



N2L Puck c.c. dir

$$F_c = ma_c, \quad a_c = \frac{v^2}{r}$$

$$T_{pm} = m \frac{v^2}{r}$$

N2L Mass g-dir

$$F_g = mg$$

$$T_{mp} - w_m = 0 \rightarrow T_{mp} = Mg$$

N2L $T_{pm} = T_{mp}$

$$\frac{v^2}{r} = Mg$$

$$M = \frac{mv^2}{gr}$$

b) CofAM conditions: isolated as surface
is frictionless

$$B_{\text{before}} = A_{\text{after}}$$

$$L_{\text{Tot}} = L'_{\text{Tot}}$$

$$L_p = L'_p$$

Treat as point particle in CEM

$$m v_i r$$

$$= m v_f r'$$

$$V_p = \frac{v_i r}{r'}$$

c) Work Energy theorem

$$W_{\text{Tot}} = \Delta K$$

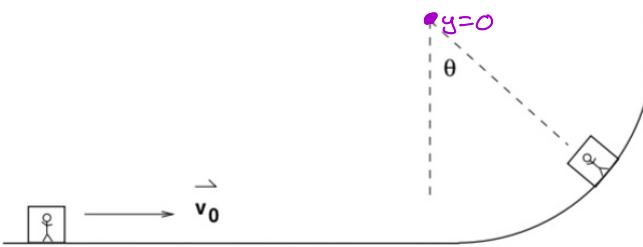
$$= \frac{1}{2} m v_f^2 - \frac{1}{2} m v_i^2$$

$$= \frac{1}{2} m \left(\frac{v^2}{r'} \right)^2 - \frac{1}{2} m v^2$$

$$= \frac{1}{2} m v^2 \left(\frac{r^2}{r'^2} - 1 \right)$$

$$W_{\text{Tot}} = \frac{1}{2} m v^2 \left(\left(\frac{r}{r'} \right)^2 - 1 \right)$$

Problem 9)



Given

- Constant speed v_0
- Block is block
- Ramp radius of R
- Frictionless surface everywhere

a) CofME

$$E_{\text{Tot}} = E'_{\text{Tot}}$$

Conditions: normal force no work \rightarrow const. cons. energy, $mgy + \frac{1}{2}mv^2 = mgy' + \frac{1}{2}mv'^2$
frictionless surface. $\frac{1}{2}v_0^2 - gR = -gR\cos\theta$

$$\theta = \cos^{-1}\left(1 - \frac{v_0^2}{2gR}\right)$$

$$1 - \frac{v_0^2}{2gR} = \cos\theta$$

b) FBD block

NZL \rightarrow -dir

$$F_{\text{Net}} = m a_x$$

$$W_{\sin\theta} = m a_x$$

$$a_x = g \sin\theta$$

$$a_x = g \sin\left(\cos^{-1}\left(1 - \frac{v_0^2}{2gR}\right)\right)$$

c) Total Acceleration

$$\vec{a}_{\text{Tot}} = \vec{a}_c + \vec{a}_x$$

$$a_c = \frac{v^2}{R}$$

$$a_x = g \sin\theta \hat{z}$$

$$a_c = \frac{v_0^2 - gR(1-\cos\theta)}{R} \hat{z}$$

$$\vec{a}_{\text{Tot}} = \frac{v_0^2 - gR(1-\cos\theta)}{R} \hat{z} + g \sin\theta \hat{u}$$

CofME

$$E_{\text{Tot}} = E'_{\text{Tot}}$$

$$mgy + \frac{1}{2}mv^2 = mgy' + \frac{1}{2}mv'^2$$

$$mg(-R) + \frac{1}{2}mv^2 = mg(-R\cos\theta) + \frac{1}{2}mv'^2$$

$$\frac{1}{2}v_0^2 - gR = -gR\cos\theta + \frac{1}{2}v'^2$$

$$\frac{1}{2}v_0^2 - gR + gR\cos\theta = \frac{1}{2}v'^2$$

$$\frac{1}{2}v_0^2 - gR(1-\cos\theta) = v'^2$$

$$v'^2 = v_0^2 - gR(1-\cos\theta)$$

d) $W_{\text{eff}} = W_{\text{ext}} + W_{\text{int}}$

$$W_{\text{ext}} = -mg \hat{j}$$

$$= -mg \cos\theta \hat{z} + mg \sin\theta \hat{u}$$

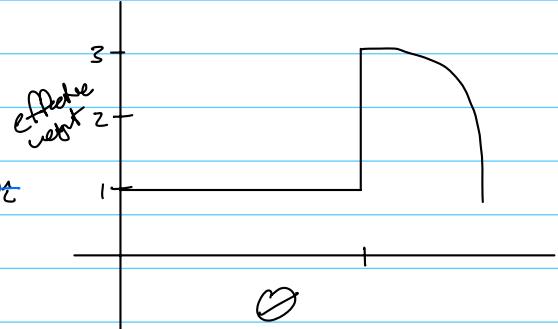
$$W_{\text{int}} = -ma_{\text{tot}}$$

$$W_{\text{eff}} = -mg \cos\theta \hat{z} + mg \sin\theta \hat{u} - m \frac{v_0^2 - gR(1-\cos\theta)}{R} \hat{z} - mg \sin\theta \hat{u}$$

$$= -mg \cos\theta \hat{z} - \frac{m(v_0^2 - gR(1-\cos\theta))}{R} \hat{z}$$

$$W_{\text{eff}} = m \left(-g \cos\theta - \frac{v_0^2 - gR(1-\cos\theta)}{R} \right) \hat{z}$$

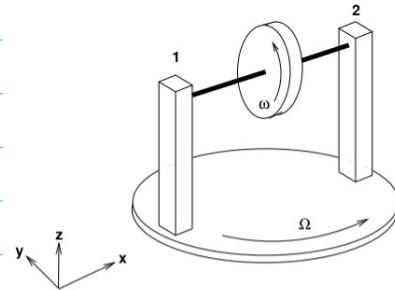
No \hat{u} component so u stays constant.



Concept Tools Chalkboard
Rotating Rotor!

$$\vec{\tau} = \frac{d\vec{L}}{dt}$$

Problem 10)



a) Corly RMR

$$L \text{ points rotate } \omega, -\dot{\omega}$$

$$L_{\text{rot}} = I\omega$$

$$= \frac{1}{2}MR^2\omega$$

$$L = -\frac{1}{2}MR^2\omega\hat{i}$$

$$\text{b) } \vec{\tau} = \frac{d\vec{L}}{dt}$$

$$\begin{aligned} \vec{\tau} &= L \frac{d\theta}{dt} \\ &= L\dot{\Omega} \\ \vec{\tau} &= \frac{1}{2}MR^2\omega\dot{\Omega}\hat{z} \end{aligned}$$

$$c) \tau_{\text{tot}} = \tau_1 + \tau_2$$

$$= -F_1 l + F_2 l \quad \text{Force}$$

$$= -\left(\frac{1}{2}mg - F\right)l + \left(\frac{1}{2}mg + P\right)l$$

$$= -\frac{1}{2}mgl + Fl + \frac{1}{2}mgl + Fl$$

$$\tau_{\text{tot}} = 2Fl \quad F = \frac{\tau_{\text{tot}}}{2l}$$

$$F = \frac{MR^2\omega\dot{\Omega}}{4l}$$

$$F_1 = \frac{1}{2}mg - \frac{MR^2\omega\dot{\Omega}}{4l}$$

$$F_2 = \frac{1}{2}mg + \frac{MR^2\omega\dot{\Omega}}{4l}$$

Pylon 2 not fine. Only exceeds
that of pylon 1

Given

- Given spin ω , mass M , radius R of turntable
- Length of axle is $2l$, does not exactly fit
- Fixed to turntable at Ω angular speed
- Ideal bearings and $\omega \gg \dot{\Omega}$
- Ω is not free precession frequency, its a given rate

Differentiating

d) $W_{\text{tot}} = \Delta K$ and w.t. velocities

being held constant results in no work done by angle moment. Noting more centrally, so central forces does no work.

\bigcirc power

$$\vec{\tau} = \frac{dL}{dt}$$

$$\vec{\tau} = I\vec{\alpha}$$

$$L = I\omega \rightarrow I = \frac{L}{\omega}$$

$$\begin{aligned} \vec{\tau} &= \frac{L}{\omega} \vec{\alpha} = \frac{L}{\omega} \cdot \frac{d^2\theta}{dt^2} \cdot \frac{d\theta}{dt} = L \frac{d\alpha}{dt} \end{aligned}$$