

MAE 158 Fall 2024

Lecture 5

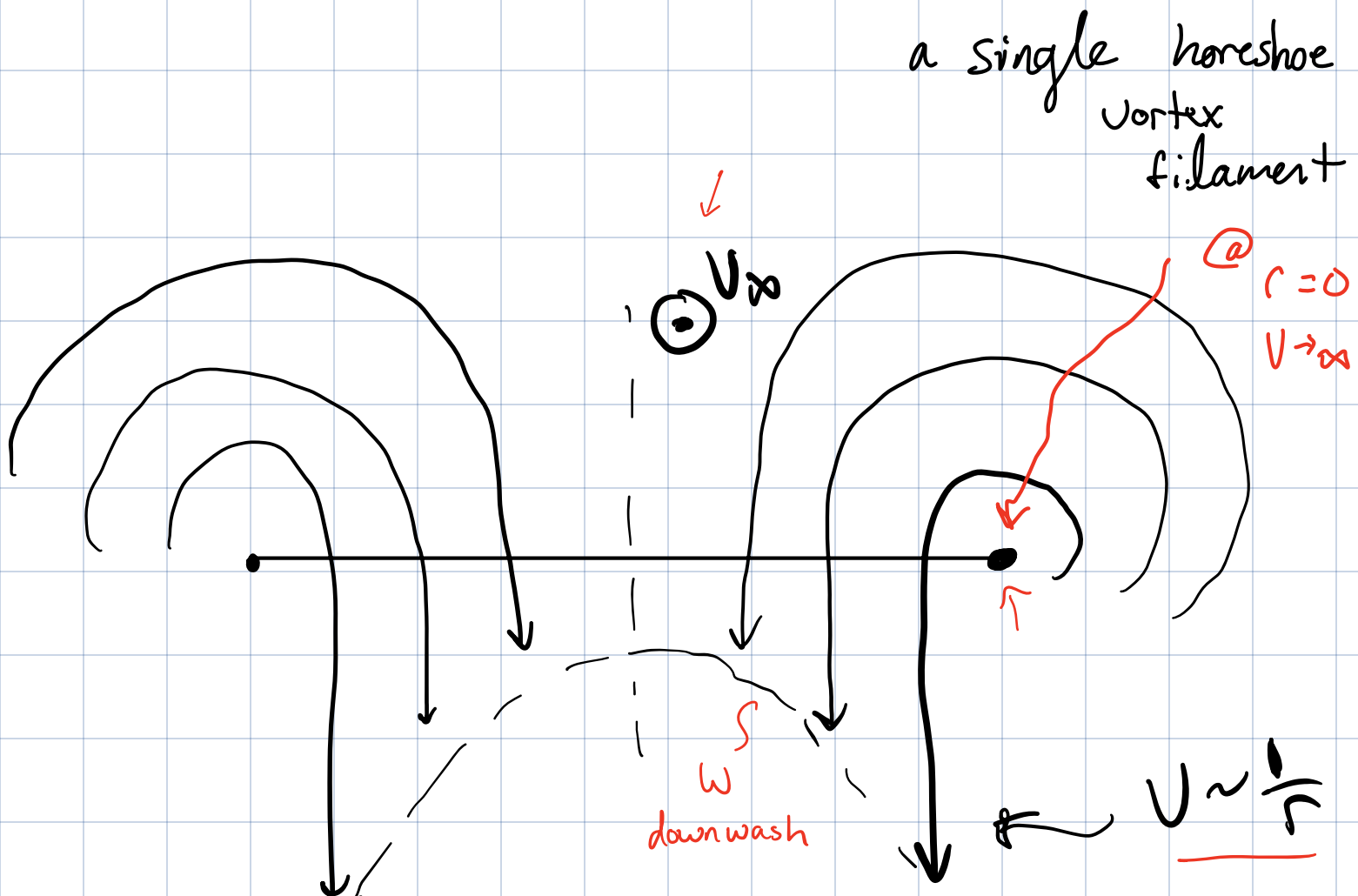
Announcements:

Today's Objectives: Induced Drag +
compressible Flow

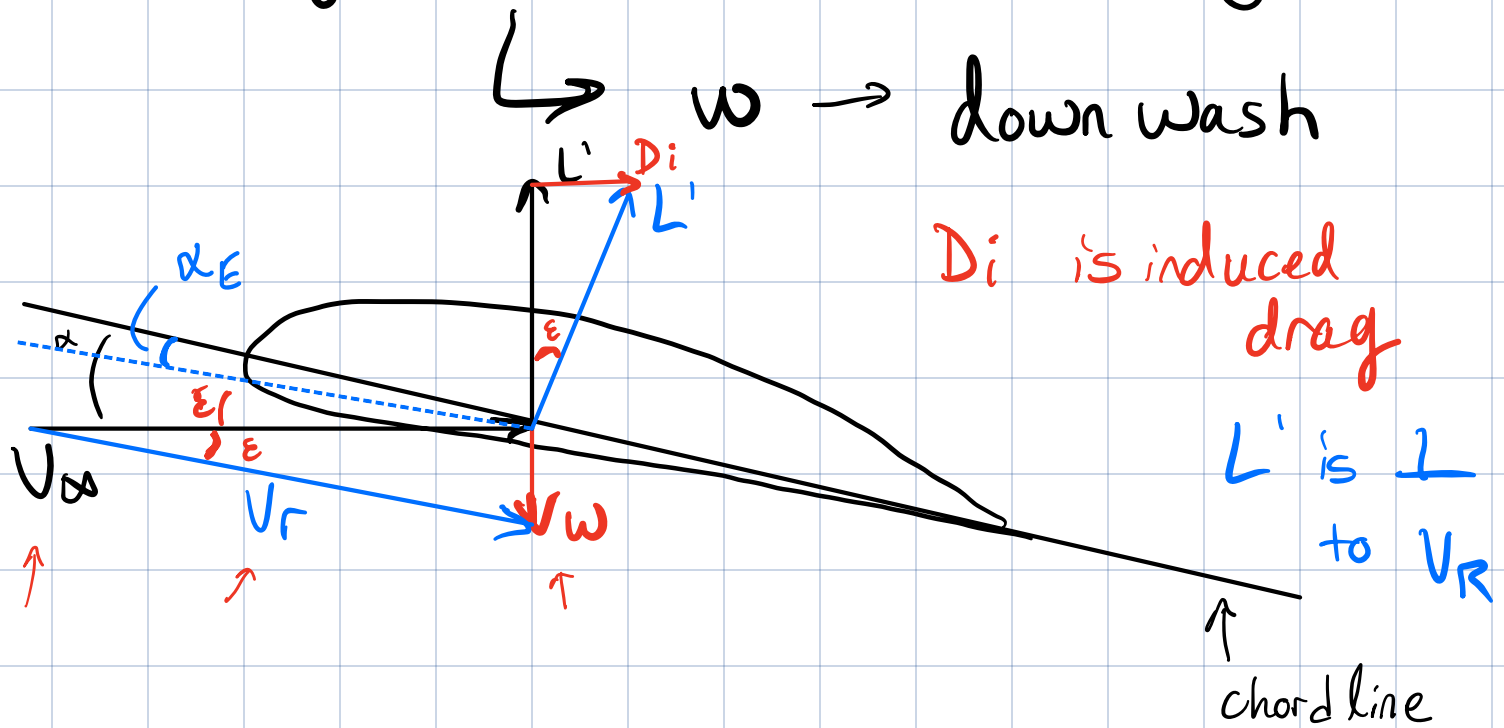
Ch. 9

Ch. 7

Last time:



velocity distribution behind wing



w Rotate $V_\infty \rightarrow V_R$
 L' rotated & D_i produced

$\alpha \equiv$ geometric angle of attack \rightarrow angle between V_∞ & chord line

$$\begin{aligned} D_i &= L' \tan \epsilon \\ \uparrow \text{induced drag} \\ &= L' \epsilon \\ &= L' \frac{w}{V_\infty} \end{aligned}$$

typically, ϵ & w are small

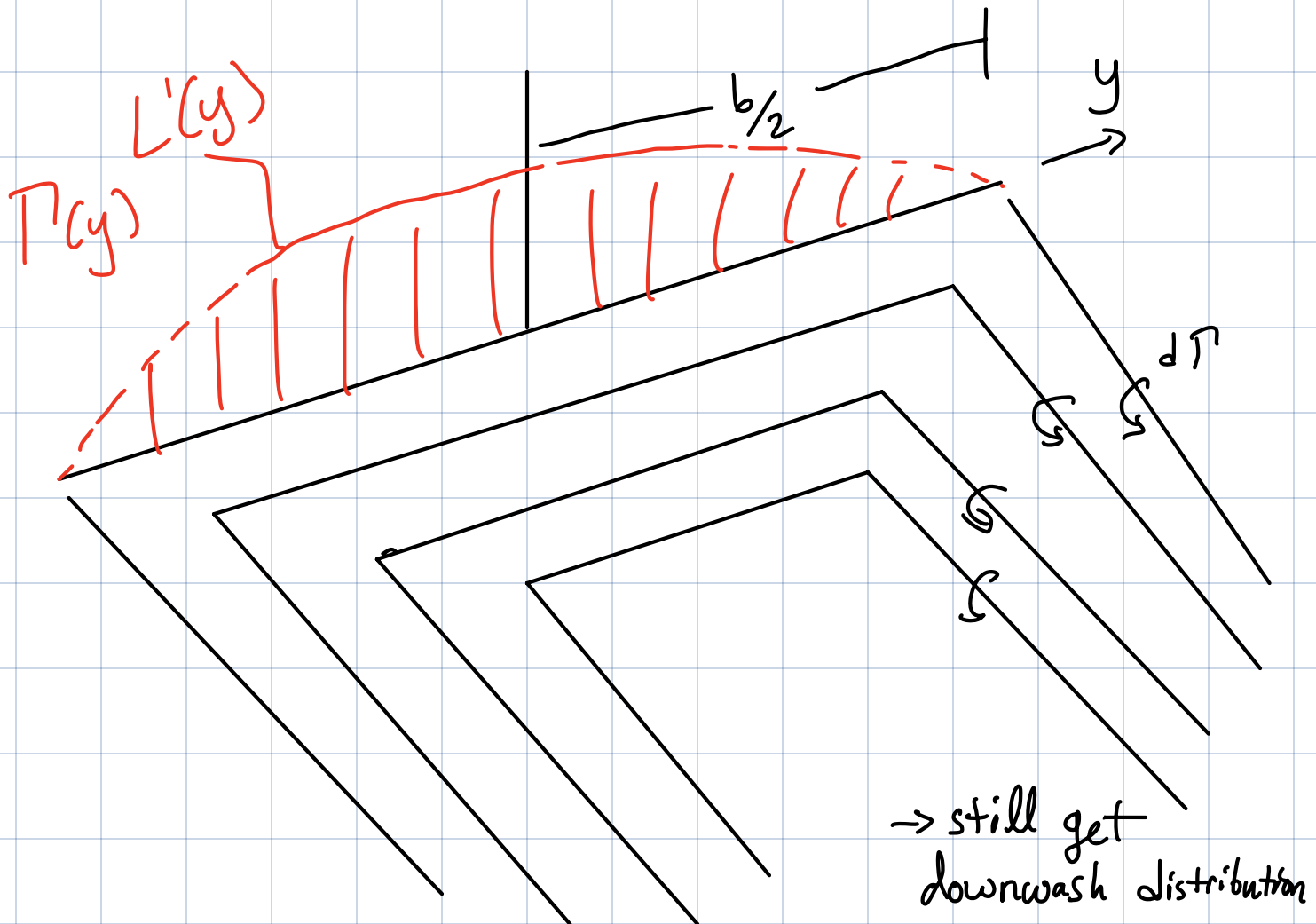
$$\epsilon \approx \frac{w}{V_\infty}$$

$\alpha_E \equiv$ effective angle of attack

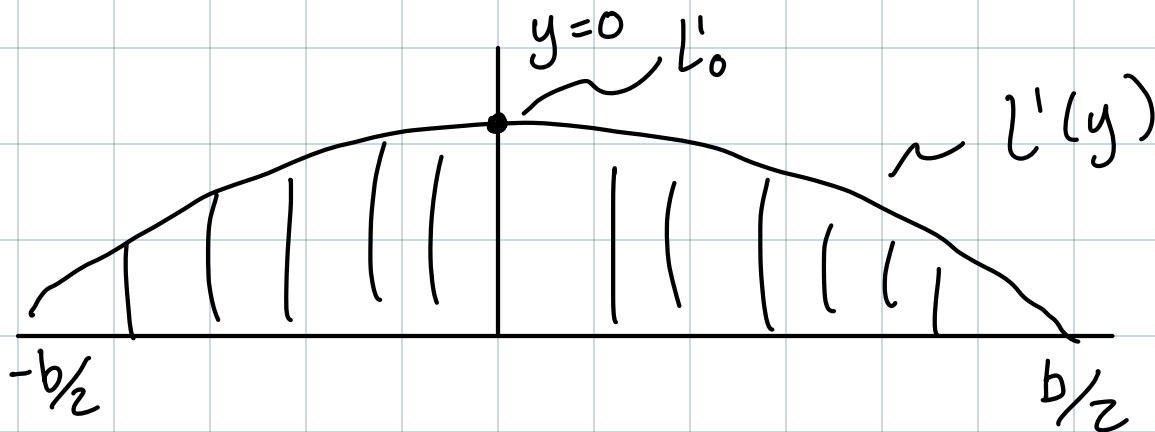
$$\begin{aligned}\alpha_E &= \alpha - \epsilon \\ &= \alpha - \omega / V_\infty\end{aligned}$$

- What is ω , the downwash?

to get ω , model the wing
as a series of horseshoe
vortices, strength $d\Gamma$



if the lift distribution is elliptical
 then the induced drag will
 be minimum for the total
 lift produced



$$L'(y) = L'_0 \sqrt{1 - (2y/b)^2}$$

$$\Gamma'(y) = \Gamma'_0 \sqrt{1 - (2y/b)^2}$$

Note, if $L'(0) = L'_0 = \rho V \Gamma'_0$

$$\begin{aligned} \text{then } L &= \int_{-b/2}^{b/2} L'(y) dy \\ &= \int_{-b/2}^{b/2} \rho V \Gamma'_0 \sqrt{1 - (2y/b)^2} dy \\ &= \frac{\pi}{4} \rho V \underset{\uparrow V_\infty}{\Gamma'_0} b \rightarrow \text{total lift} \end{aligned}$$

Solve for w (downwash) of elliptical lift distribution

$$D_i = L' \frac{w}{V_\infty}$$

use Biot-Savart Law

$$dw = -\frac{d\Gamma}{4\pi} \frac{1}{y_0 - y}$$

y_0 is a location where we want to determine downwash that is produced by a filament @ y

$$\begin{aligned} w(\underline{y_0}) &= -\frac{1}{4\pi} \int_{-b/2}^{b/2} \frac{\frac{d\Gamma}{dy}}{y_0 - y} dy \\ &= -\frac{\Gamma_0}{\pi b^2} \int_{-b/2}^{b/2} \frac{y}{\sqrt{1 - (2y/b)^2}} \frac{dy}{y_0 - y} \\ &= -\frac{\Gamma_0}{2b} \leadsto \text{const.} \end{aligned}$$

elliptical lift distribution:

- ω is const, along the span
- ε is const, along the span

$$D_i' = L \omega / V_\infty = L \varepsilon$$

$$C_{Di} = C_L \omega / V_\infty$$

Recall that $\varepsilon = \frac{\omega}{V_\infty} = \frac{\Gamma_0}{2b V_\infty}$

$$L = \Gamma_0 \frac{\pi}{4} \rho V_\infty b$$

then $\varepsilon = \frac{L}{\pi \frac{1}{2} \rho V_\infty^2 b^2}$

note $C_L = \frac{L}{\frac{1}{2} \rho V_\infty^2 S}$

then $\varepsilon = \frac{C_L}{\pi R}$

$$A = \frac{b^2}{S_{REF}} \equiv \text{Aspect Ratio Wing}$$

$$\underline{C_{Di}} = C_L \cdot \epsilon = \frac{C_L^2}{\pi A}$$

Lift

geometrical factor of wing

elliptical lift distribution

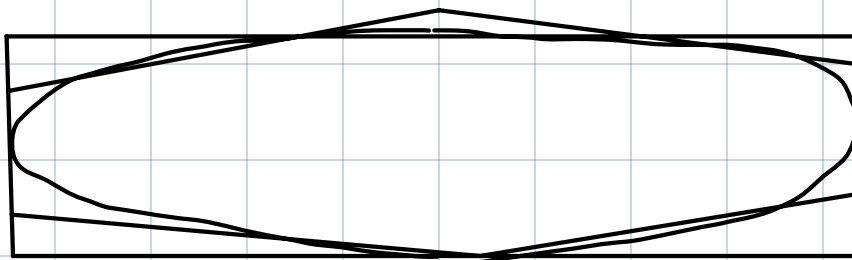
$$D_i = \frac{1}{2} \rho V_\infty^2 \cdot \overset{\substack{\uparrow \\ S_{REF}}}{S_{REF}} C_{Di}$$

$$= \frac{1}{2} \rho V_\infty^2 S \left(\frac{C_L^2}{\pi A} \right) \quad \leftarrow L = \frac{1}{2} \rho V^2 S C_L$$

$$= \frac{1}{\pi q} \cdot \left(\frac{L^2}{b^2} \right)$$

\uparrow
 $\frac{1}{2} \rho V^2$

if distribution of Lift is Not elliptical



$$C_{Di} = \frac{C_L^2}{\pi R e}$$

$e \equiv$ span
efficiency
factor
↑
shewelle \rightarrow "u"

if elliptical $\rightarrow e = 1$

if not elliptical $\rightarrow e < 1$

for $C_{Di} \rightarrow$ must use C_L
of the wing!
not,
 C_L of the
airfoil

if you only have C_L of the
airfoil, must convert to
 C_L in 3D \rightarrow know what
 R will be

Recal: $\alpha_E = \alpha - \epsilon$

\uparrow effective AoA \nwarrow geometric $\uparrow \frac{C_L}{\pi R}$

consider lift curve slope:

2D: $C_l = a_0 \cdot \alpha + \text{const}$

$\uparrow 2\pi\eta$ (α is in rad)

in 3D: $C_L = a_0 \cdot \alpha_E$

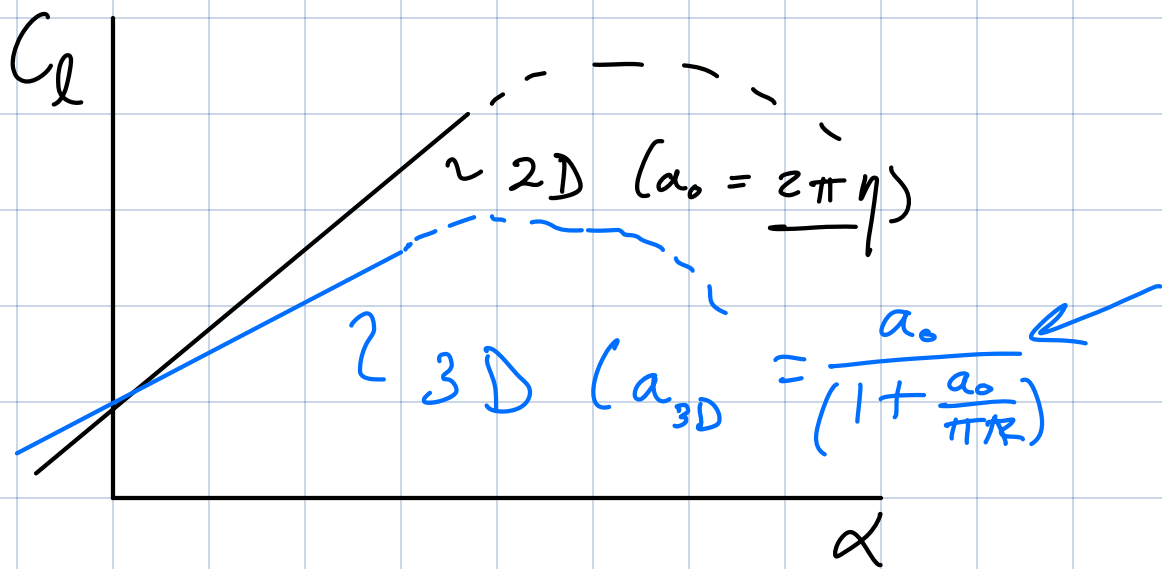
$= a_0 \left(\alpha - \frac{C_L}{\pi R} \right)$

$\uparrow 2\pi\eta$

thus,

$C_L = \frac{a_0 \cdot \alpha}{\left(1 + \frac{a_0}{\pi R}\right)} \uparrow \text{geometric } \alpha$

$a_{3D} = \frac{dC_L}{d\alpha} = \frac{a_0}{\left(1 + \frac{a_0}{\pi R}\right)}$



MUST use C_L when $\rightarrow D_i$

$$D_{\text{total}} = D_p + D_i + D_c$$

\swarrow skin fric + pressure drag $\uparrow C_{Di} = \frac{C_L^2}{\pi R e}$ \nwarrow compressibility

compressible flow

$P_1 \neq P_2 \rightarrow$ pressure changes \rightarrow change P

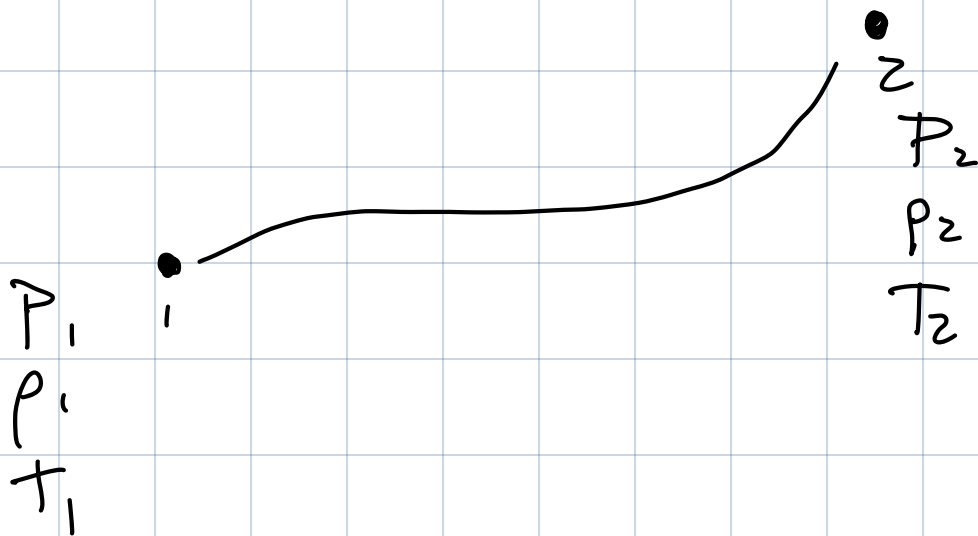
$M > 0.3 \rightarrow$ condition for

compressible Flow

$$\text{Mach \#} = \frac{V}{a} \quad a \rightarrow \text{speed of sound}$$

- Relations for compressible Flow between ρ, V, P, M

- considering a process from 1 to 2



a few concepts for different processes to consider

- adiabatic: No heat is added or taken

- Reversible: No friction is dissipated

- Isentropic: adiabatic & Reversible

most aerodynamic flows, if outside of BL, Reversible, & adiabatic

thus; can apply isentropic gas laws

$$\left(\frac{P_2}{P_1} \right) = \left(\frac{\rho_2}{\rho_1} \right)^\gamma$$

γ = ratio of specific heats

$\gamma = 1.4$ if air

equation of state $\rho = \frac{P}{RT}$

then
$$\frac{P_2}{P_1} = \left(\frac{P_2 / RT_2}{P_1 / RT_1} \right)^\gamma = \left(\frac{T_2}{T_1} \right)^{\frac{\gamma}{\gamma-1}}$$

also:
$$\frac{\rho_2}{\rho_1} = \left(\frac{T_2}{T_1} \right)^{\frac{1}{\gamma-1}}$$

→ established Relation
between ρ, P, T

how about T, V ?

energy equation

$$C_p \cdot T + \frac{V^2}{2} = \text{const}$$

\uparrow specific heat capacity

$$= C_p \cdot T_t$$

$$\equiv 1004.7 \frac{\text{J}}{\text{kgK}} \quad \text{for air}$$

$$= 6006 \frac{\text{lb-ft}}{\text{slug R}} \quad \text{for air}$$

how about P, V ? $\rho \neq \text{const}$

Euler equation $dp = -\rho V dV$

denote $()_T \equiv$ stagnation property

where $V=0$

Sub in isentropic Relation $\rho = \rho_t \left(\frac{P}{P_t} \right)^{1/\gamma}$

$$\text{thus } dp = - \left(\frac{P}{P_t} \right)^{1/\gamma} \rho_t V dV$$

$$\text{integrate } 0 = \int_{P_t}^P dp \left(\frac{P}{P_t} \right)^{1/\gamma} \frac{1}{\rho_t} + \int_0^V V dV$$

$$\hookrightarrow \frac{\gamma}{\gamma-1} \left(\frac{P_t}{P_t} \right) \left(\frac{P}{P_t} \right)^{\frac{\gamma-1}{\gamma}} + \frac{V^2}{2} = \frac{\gamma}{\gamma-1} \frac{P_t}{P_t}$$

compressible Bernoulli equation

get Relations between M & stagnation, static properties

$$M = V/a$$

$$a = \sqrt{\gamma R T}$$

$$\text{-or- } a = \sqrt{\gamma \frac{P}{\rho}}$$

standard sea level
air

$$T = 288 \text{ K}$$

$$R = 287 \text{ Nm/kgK}$$

$$a = \sqrt{(1.4)(287 \frac{\text{Nm}}{\text{kgK}})(288 \text{ K})}$$

$$= 340 \text{ m/s @ sea level}$$

$$= 1116 \text{ ft/s}$$

$$V = a \cdot M = \sqrt{\gamma R T} \cdot M$$

go back to
energy equation

$$C_p T_1 + \frac{V_1^2}{2} = C_p T_2 + \frac{V_2^2}{2}$$

Relate stagnation to static
properties
($V=0$) \leadsto ()_T

$$C_p = \frac{\gamma R}{\gamma - 1}$$

$$a^2 = \gamma R T_1$$

$$\frac{T_t}{T_1} = 1 + \frac{V_1^2}{2c_p T_1}$$

$$= 1 + \frac{\gamma-1}{2} M_1^2$$

plug in isentropic flow relations

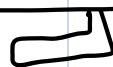
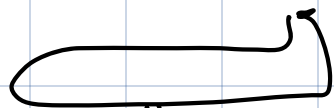
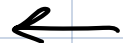
$$\frac{P_t}{P_0} = \left(1 + \frac{\gamma-1}{2} M_1^2\right)^{\gamma/(\gamma-1)}$$

$$\frac{P^*}{P_1} = \left(1 + \frac{\gamma-1}{2} M_1^2\right)^{1/(\gamma-1)}$$

$$M_1 = \sqrt{\frac{2}{\gamma-1} \left[\left(\frac{P_t}{P_1} \right)^{\frac{\gamma-1}{\gamma}} - 1 \right]}$$

↳ get M_1 knowing local
stagnation (total) pressure
& static pressure

M, V



pitot tube

$\hookrightarrow P_t$



What is $M, ?$