# MAE 5803 Nonlinear Control Systems Homework #4 (Solutions)

### a. Courtesy of Casey Clark

A single-degree of freedom underwater vehicle motion under viscous drag can be modeled as:

$$m\ddot{x} + (\alpha_1 + \alpha_2 \cos^2 x)|\dot{x}| \dot{x} = u + d \qquad (1)$$

Assigned: Feb 28, 2017

Due: Mar 21, 2017

with bounds

$$4 \le \alpha_1 \le 6$$
$$1 \le \alpha_2 \le 2$$
$$|d| \le 1$$

#### Part a

For m = 1 equation (1) simplifies

$$\ddot{x} = -(\alpha_1 + \alpha_2 \cos^2 x)|\dot{x}|\dot{x} + d + u$$

A switching controller can be designed for an equation in the form  $\ddot{x} = f + u$  using the control law

$$u = \hat{u} - ksgn(s)$$
$$\hat{u} = -\hat{f} + \ddot{x}_d - \lambda \dot{\tilde{x}}$$
$$k = F + \eta$$

where the sliding line is defined as

$$s = \dot{\tilde{x}} + \lambda \tilde{x}$$

with tracking error  $\tilde{x}$ 

$$\tilde{x} = x - x_d$$
$$\dot{\tilde{x}} = \dot{x} - \dot{x}_d$$

F is bounded by

$$\left| f - \hat{f} \right| \leq F$$

with

$$f = -(\alpha_1 + \alpha_2 \cos^2 x)|\dot{x}|\,\dot{x} + d$$

An appropriate selection of parameters for  $\hat{f}$  can be made by substituting the mean of each constant into f

$$\alpha_1 = 4 + \frac{6-4}{2} = 5$$

$$\alpha_2 = 1 + \frac{2-1}{2} = 1.5$$

$$d = -1 + \frac{1-(-1)}{2} = 0$$

Yielding

$$\hat{f} = -(5 + 1.5\cos^2 x)|\dot{x}|\dot{x}$$

The values of the parameters for F can be chosen to be the difference between the max and mean bound

$$\alpha_1 = 6 - 5 = 1$$
 $\alpha_2 = 2 - 1.5 = 0.5$ 
 $d = 1 - 0 = 1$ 

Yielding

$$F = (1 + 0.5\cos^2 x)\dot{x}^2 + 1$$

Substituting F into k yields

$$k = (1 + 0.5\cos^2 x)\dot{x}^2 + 1 + \eta$$

Substituting  $\hat{f}$  into  $\hat{u}$ 

$$\hat{u} = (5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x} + \ddot{x}_d - \lambda\dot{\tilde{x}}$$

Finally substitute  $\hat{u}$  into u

$$u = (5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x} + \ddot{x}_d - \lambda\dot{\tilde{x}} - ksgn(s)$$

### Part b

The system response was simulated to track  $x_d(t) = 2 \sin t$  using the above controller and gain with

$$\alpha_1(t) = 5 + \cos t$$

$$\alpha_2(t) = 1 + |\sin t|$$

$$d(t) = \cos 1.3t$$

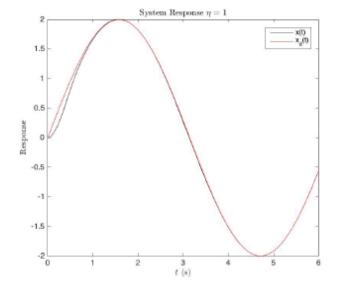
$$\lambda = 4$$

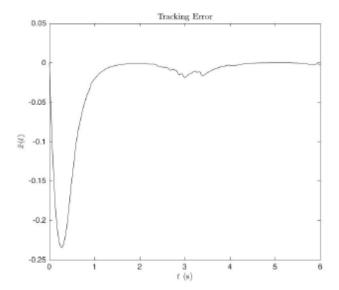
Note that

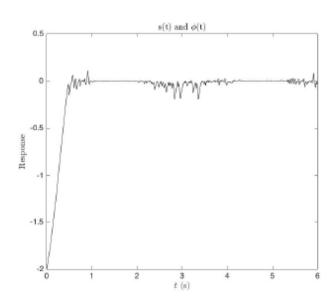
$$\dot{x}_d(t) = 2\cos t$$

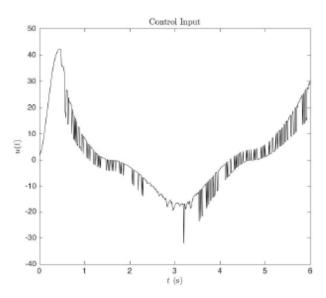
$$\ddot{x}_d(t) = -2\sin t$$

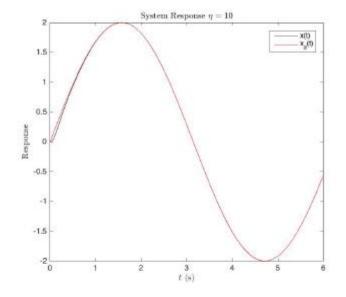
There is noticeable chattering in the control responses for both values of  $\eta$ . An increase in  $\eta$  appears to decrease transient response time of x(t) as well decrease the tracking error.

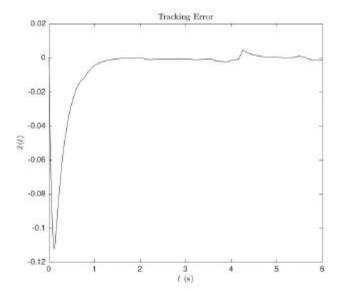


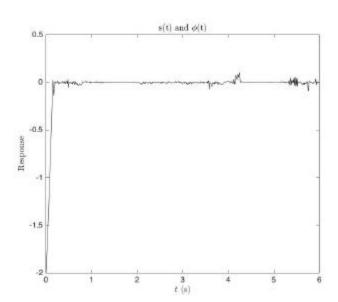


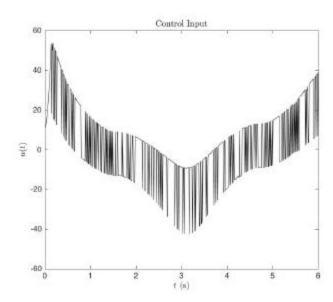












## Part c

A sliding controller can be designed for an equation in the form  $\ddot{x} = f + u$  using the control law

$$u = \hat{u} - (k - \dot{\phi})sat\left(\frac{s}{\phi}\right)$$
  
 $\hat{u} = -\hat{f} + \ddot{x}_d - \lambda \dot{\tilde{x}}$   
 $\dot{\phi} + \lambda \phi = k(x_d)$   
 $k(x_d) = F(x_d) + \eta$ 

Use the same  $\hat{u}$ ,  $\hat{f}$ , and k used in the switching controller

$$\hat{u} = (5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x} + \ddot{x}_d - \lambda\dot{\tilde{x}}$$
 
$$\hat{f} = -(5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x}$$
 
$$k = F + \eta$$

with tracking error  $\tilde{x}$ 

$$\tilde{x} = x - x_d$$

$$\dot{\tilde{x}} = \dot{x} - \dot{x}_d$$

and the same F used in the switching controller

$$F = (1 + 0.5\cos^2 x)\dot{x}^2 + 1$$

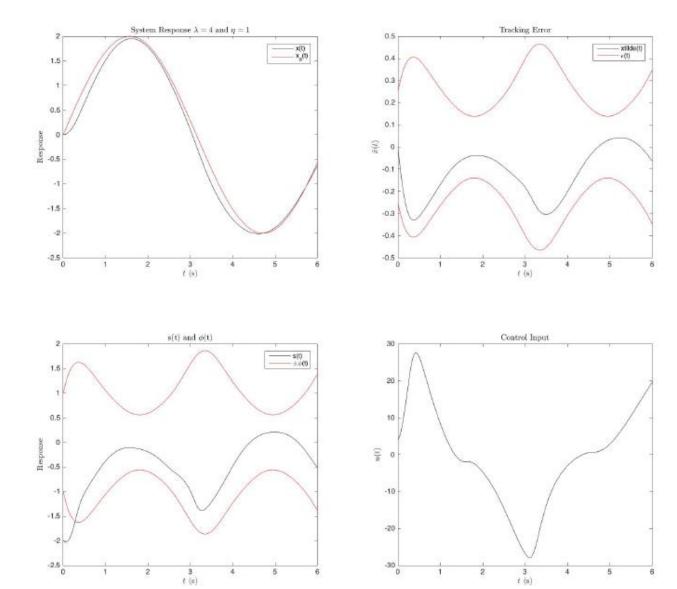
Now  $k(x_d)$  can be evaluated as

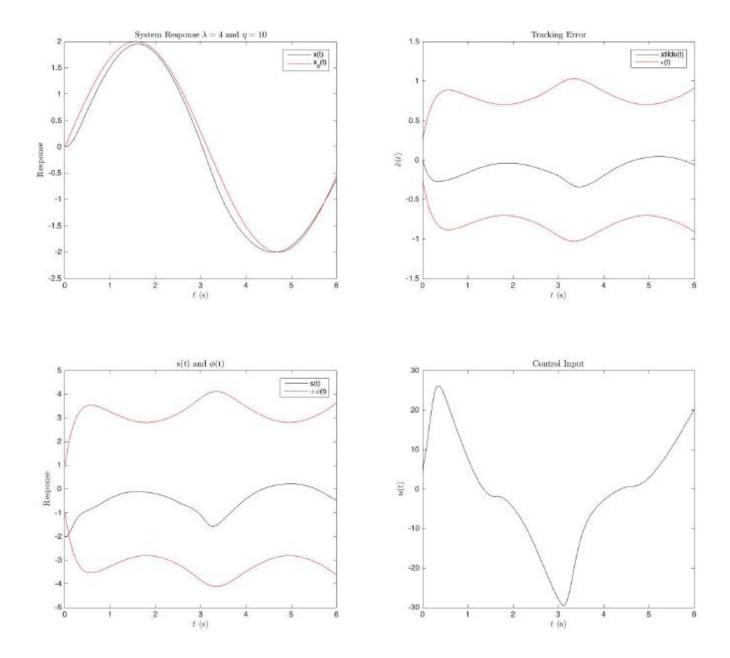
$$k(x_d) = (1 + 0.5\cos^2 x_d)\dot{x_d}^2 + 1 + \eta$$

Yielding the controller

$$u = (5 + 1.5\cos^2 x)|\dot{x}| \, \dot{x} + \ddot{x}_d - \lambda \dot{\tilde{x}} - \left[ (1 + 0.5\cos^2 x)\dot{x}^2 - (1 + 0.5\cos^2 x_d)\dot{x}_d^2 + \lambda \phi \right] sat\left(\frac{s}{\phi}\right)$$

Part d





As  $\lambda$  increases, the tracking error becomes less and less, also the width of boundary layer becomes smaller which shows that sliding variable for large  $\lambda$  stays closer to s=0 than smaller  $\lambda$ .

As  $\eta$  increases the controller acts faster in bringing the sliding variable to s=0 line, hence the tracking is done in shorter time.

Based on theory 
$$|\tilde{x}| \leq \frac{\phi}{\lambda^{n-1}} \to |\tilde{x}| \leq \frac{\phi}{\lambda^1}$$

### Part e

A modification to the switching controller can be made to account for a time-varying m bounded by

$$1 \leq m \leq 2$$

Now

$$\ddot{x} = -\frac{1}{m}(\alpha_1 + \alpha_2 \cos^2 x)|\dot{x}|\,\dot{x} + \frac{1}{m}d + \frac{1}{m}u$$

with

$$f = -\frac{1}{m}(\alpha_1 + \alpha_2 \cos^2 x)|\dot{x}|\,\dot{x}$$
 
$$b = \frac{1}{m}$$

To account for varying mass define  $\beta$  as

$$\beta = \sqrt{\frac{b_{max}}{b_{min}}} = \sqrt{2}$$
$$\beta^{-1} \le \frac{b}{\hat{b}} \le \beta$$

where  $\hat{b}$  is the estimate of b defined as the geometric mean of b

$$\hat{b}=\sqrt{b_{min}b_{max}}=rac{\sqrt{2}}{2}$$
 
$$\hat{m}=rac{1}{\hat{b}}=\sqrt{2}$$

A sliding controller can be designed for an equation in the form  $\ddot{x} = f + bu$  using the control law

$$u = \hat{u} - (k - \dot{\phi})sat\left(\frac{s}{\phi}\right)$$
$$\hat{u} = -\hat{f} + \ddot{x}_d - \lambda \dot{\ddot{x}}$$
$$\dot{\phi} + \lambda \phi = k(x_d)$$
$$k(x_d) = F(x_d) + \eta$$

with tracking error  $\tilde{x}$ 

$$\tilde{x} = x - x_d$$

$$\dot{\tilde{x}} = \dot{x} - \dot{x}_d$$

Modifying  $\hat{f}$  and  $\hat{u}$  to account for varying mass

$$\hat{f} = -\frac{1}{1.4}(5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x}$$

$$\hat{u} = \frac{1}{1.4}(5 + 1.5\cos^2 x)|\dot{x}|\,\dot{x} + \ddot{x}_d - \lambda\dot{x}$$

The gain k is modified to account for the varying mass by redefining it as

$$k = \beta(F+\eta) + (\beta-1) \Big| \ddot{x}_d - \lambda \dot{\tilde{x}} \Big|$$

Modify F to account for the varying mass

$$F = \frac{1}{1.4} \left( (1 + 0.5 \cos^2 x) \dot{x}^2 + 1 \right)$$

## Part f

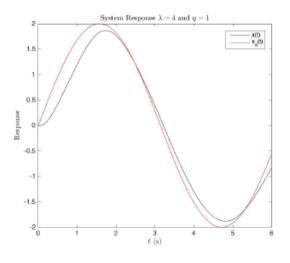
The response was simulated for

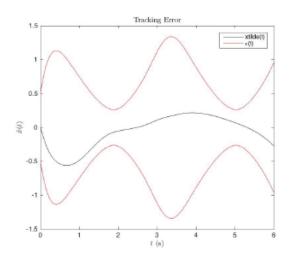
$$\alpha_1(t) = 5 + \cos t$$

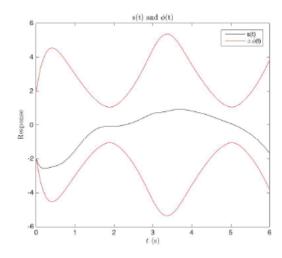
$$\alpha_2(t) = 1 + |\sin t|$$

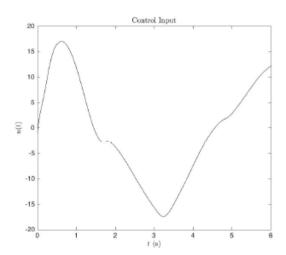
$$d(t) = \cos 1.3t$$

$$m(t) = 2 - |\cos 1.5t|$$









2. Courtesy of Shingo Kunito

Courtesy of	Sningo Kunito
2) 5	i + 0,(4) /2/2 + 02(4) 23 cos22 = 5 i + 11
	14 4 4 5
	1×1(+)151, -15×25
01)	et v= 5 il 111
	( 2 2 2 - 2d (trucking error)
	let $S = \tilde{\chi} + \chi \tilde{\chi}$
	S = 2 - 2d + 2 2
	5 - 20 + 2 2
where	x: Y = a,(t) 12/22 - x2(t) x cos2x
let	f a, (6) x12 - x2 (4) x3 cos22
	f.= - &, 1x1 x2 - &2 28 (0522
	V= x - ksqn(s)
	2 = -f + 2/1 - 2 2
	Assuming R. = 0, Re = 2
	1 Sur 1 1 1 1 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1
	=> 0 2x3cos2x + xd + x2
	V= 223 cos22+2d+22 - lesgn (s)
	=> $\dot{s} = 2x^3\cos 2x - \alpha \cdot  x \dot{x}^2 - \alpha_2 x^3\cos 2x - kgyn (s)$
	F= 17- £1
	$=  \hat{\alpha}_1 \chi \dot{\chi}^2 + \hat{\alpha}_2\chi^3\cos 2\chi - \chi_1 \chi \dot{\chi}^2 - \alpha_2\chi^3\cos 2\chi$
	= 1 22 cos2x - x, 121 22 - 02 23 (05 22 )
	E
	F = 12/2 + 3 2 12/ 100522/
	L = E 10
	k= F+7

