

Topology optimization design for total sound absorption in porous media

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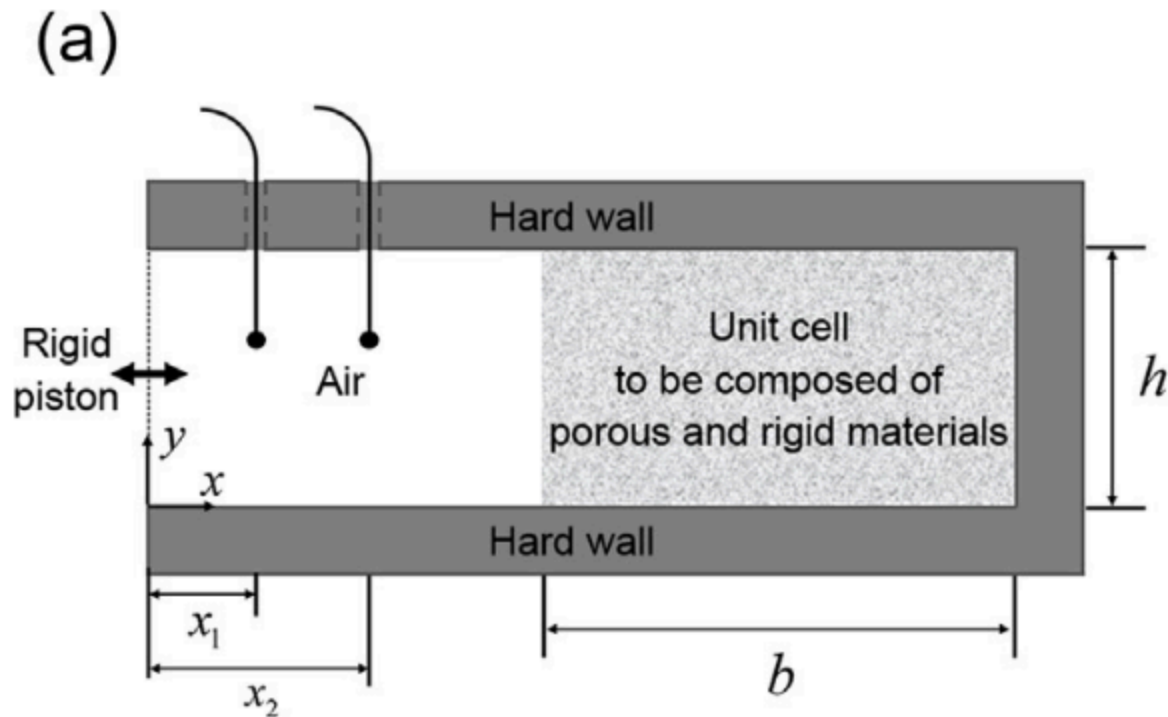
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Macroscopic topology optimization, rather than multi-scale topology optimization.

Learn the numerical calculation methods of sound absorption coefficients.

The energy of sound is divided into three parts: reflected, transmitted, and dissipated.

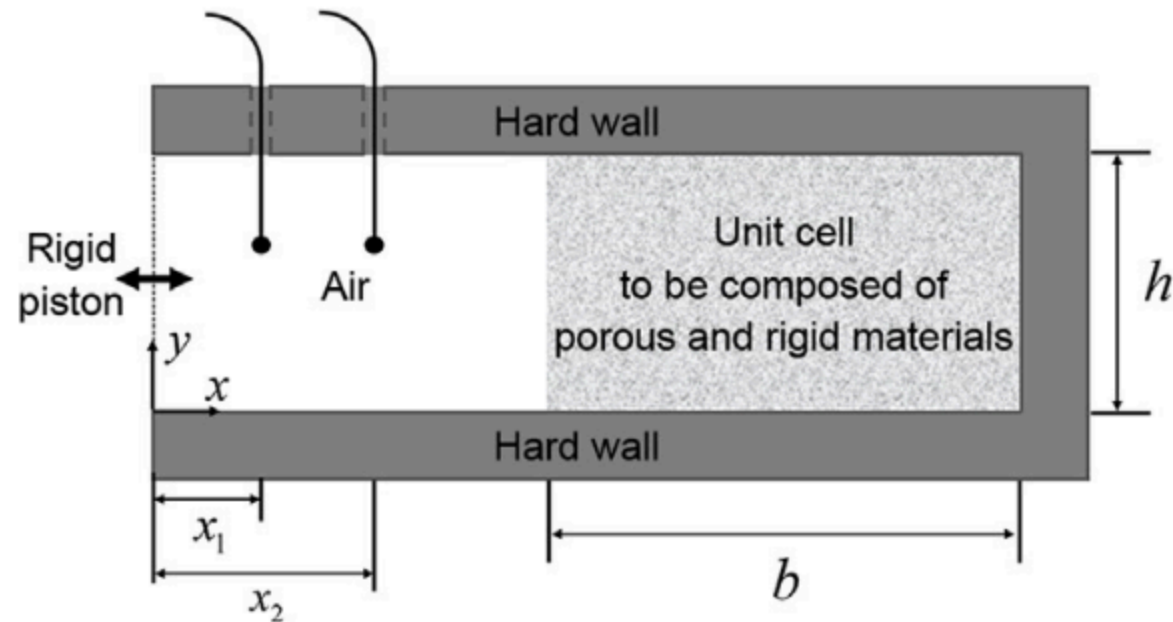


Enclosed by an acoustically "hard wall," with no transmitted portion.

$$P(x) = Ae^{-ikx} + Be^{ikx}, \quad R = \frac{B}{A}, \quad \alpha = 1 - |R|^2$$

$$R(\chi; f) = \frac{-p_2 \cdot \exp(-jk_a x_1) + p_1 \cdot \exp(-jk_a x_2)}{p_2 \cdot \exp(jk_a x_1) - p_1 \cdot \exp(jk_a x_2)}.$$

(a)



To calculate the sound absorption coefficient, it is necessary to solve the equations to determine p_1 and p_2 . The equation for sound propagation is

$$\frac{\partial^2 p}{\partial t^2} = c^2(x) \nabla^2 p.$$

The steady-state acoustic pressure p of an acoustic domain is governed by

$$\rho^{-1} \nabla^2 p + \omega^2 \kappa^{-1} p = 0,$$

where the symbols ρ and κ denote the density and bulk modulus of the acoustic medium filled in the domain, respectively.

By applying the variational principle and the divergence theorem to Eq. (1), the weighted residual integral equation (i.e., the weak integral form) of the scalar Helmholtz equation can be written as [29]:

$$\int_{\Omega} \rho^{-1} (\nabla \tilde{p} \cdot \nabla p) d\Omega - \omega^2 \int_{\Omega} \kappa^{-1} \tilde{p} p d\Omega + \int_{\partial\Omega} j\omega \tilde{p} (\mathbf{v} \cdot \mathbf{n}) d\Gamma = 0, \quad (2)$$

where the symbols \tilde{p} and \mathbf{v} denote the weighting acoustic pressure field and the particle velocity of the acoustic medium, respectively. In Eq. (2), the unit vector \mathbf{n} is defined to be outward normal to the boundary $\partial\Omega$ of the acoustic domain Ω .

Substituting Eqs. (3a,b) into the element version of Eq. (2) yields the following matrix–vector form equation in the e th element Ω_e :

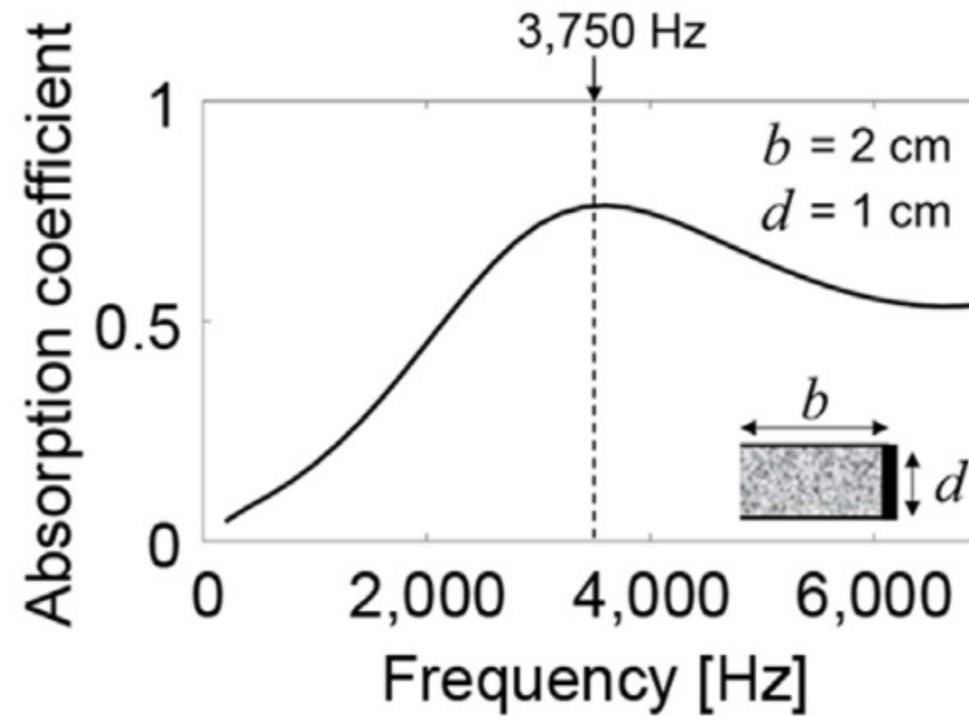
$$[\mathbf{K}_e - \omega^2 \mathbf{M}_e] \mathbf{P}_e = \mathbf{F}_e, \quad (4a)$$

where the element stiffness matrix \mathbf{K}_e and mass matrix \mathbf{M}_e are defined as:

$$\mathbf{K}_e = \int_{\Omega_e} \rho_e^{-1} \mathbf{B}_e^T \mathbf{B}_e d\Omega, \quad \mathbf{M}_e = \int_{\Omega_e} \kappa_e^{-1} \mathbf{N}_e^T \mathbf{N}_e d\Omega. \quad (4b,c)$$

In Eq. (4a), \mathbf{F}_e is the nodal acoustic force vector in the element. The symbols ρ_e and κ_e in Eqs. (4b,c) are the density and bulk modulus of an acoustic medium filling the e th element.

Two target materials in unit cells: a porous material and a rigid material





0



0



0



0

$$\min_{\chi} F(\chi) = \min_{\chi} \left[\frac{1}{N} \sum_{e=1}^N \chi_e (1 - \chi_e) \right],$$

subject to $C(\chi; f_i) = 1 - \alpha(\chi; f_i) \leq \varepsilon$ with $i = 1, 2, \dots, N_t$

$$\frac{1}{\rho_e(\chi_e)} = (\chi_e)^p \left\{ \frac{1}{\rho_{porous}} - \frac{1}{\rho_{rigid}} \right\} + \frac{1}{\rho_{rigid}},$$

$$\frac{1}{\kappa_e(\chi_e)} = (\chi_e)^q \left\{ \frac{1}{\kappa_{porous}} - \frac{1}{\kappa_{rigid}} \right\} + \frac{1}{\kappa_{rigid}},$$

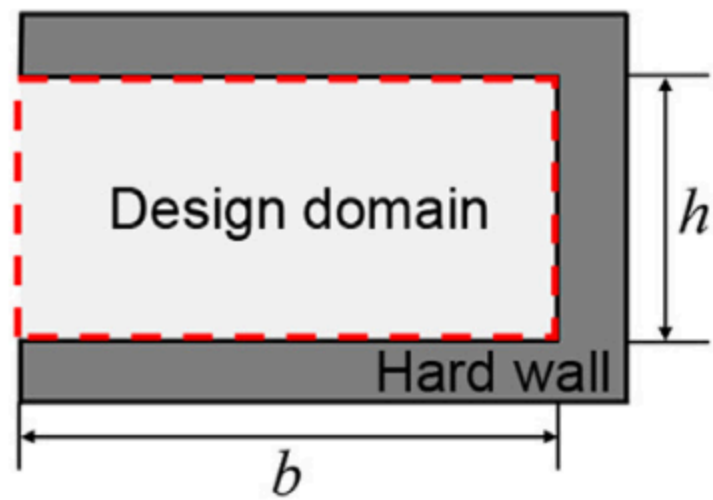
$$\frac{\partial \alpha(\boldsymbol{\chi}; f_i)}{\partial \chi_e} = -2 |R(\boldsymbol{\chi}; f_i)| \cdot \frac{\partial |R(\boldsymbol{\chi}; f_i)|}{\partial \chi_e}, \quad (13a)$$

$$\begin{aligned} \frac{\partial R(\boldsymbol{\chi}; f_i)}{\partial \chi_e} = & \frac{\left\{ -\frac{\partial p_2}{\partial \chi_e} \cdot \exp(-jk_a x_1) + \frac{\partial p_1}{\partial \chi_e} \cdot \exp(-jk_a x_2) \right\} \{p_2 \cdot \exp(jk_a x_1) - p_1 \cdot \exp(jk_a x_2)\}}{\{p_2 \cdot \exp(jk_a x_1) - p_1 \cdot \exp(jk_a x_2)\}^2} \\ & - \frac{\{-p_2 \cdot \exp(-jk_a x_1) + p_1 \cdot \exp(-jk_a x_2)\} \left\{ \frac{\partial p_2}{\partial \chi_e} \cdot \exp(jk_a x_1) - \frac{\partial p_1}{\partial \chi_e} \cdot \exp(jk_a x_2) \right\}}{\{p_2 \cdot \exp(jk_a x_1) - p_1 \cdot \exp(jk_a x_2)\}^2}, \end{aligned} \quad (13b)$$

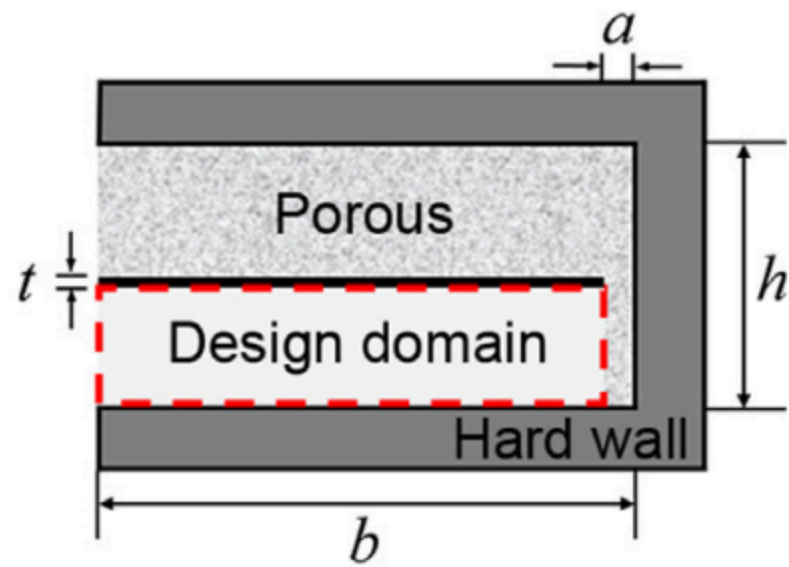
where the expression $\partial p / \partial \chi_e$ in Eq. (13b) can be obtained using the system equation in Eq. (5) as:

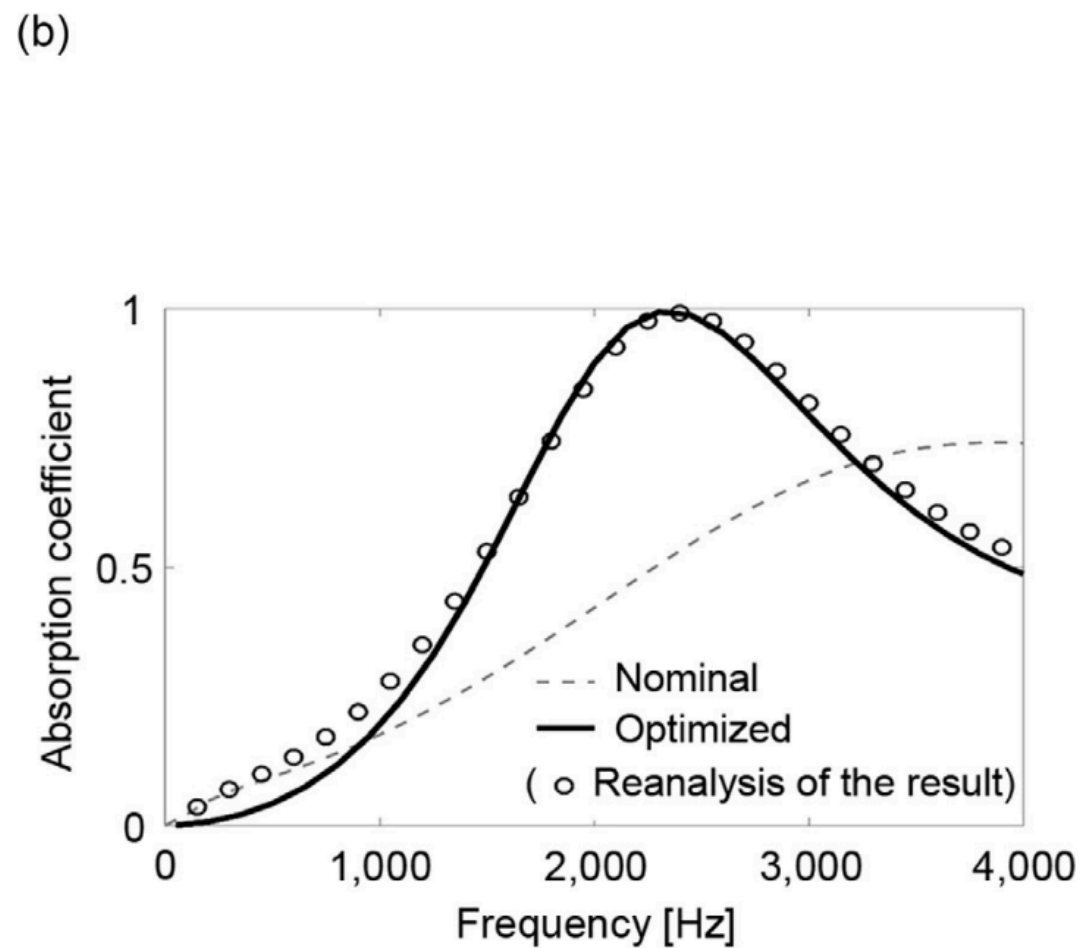
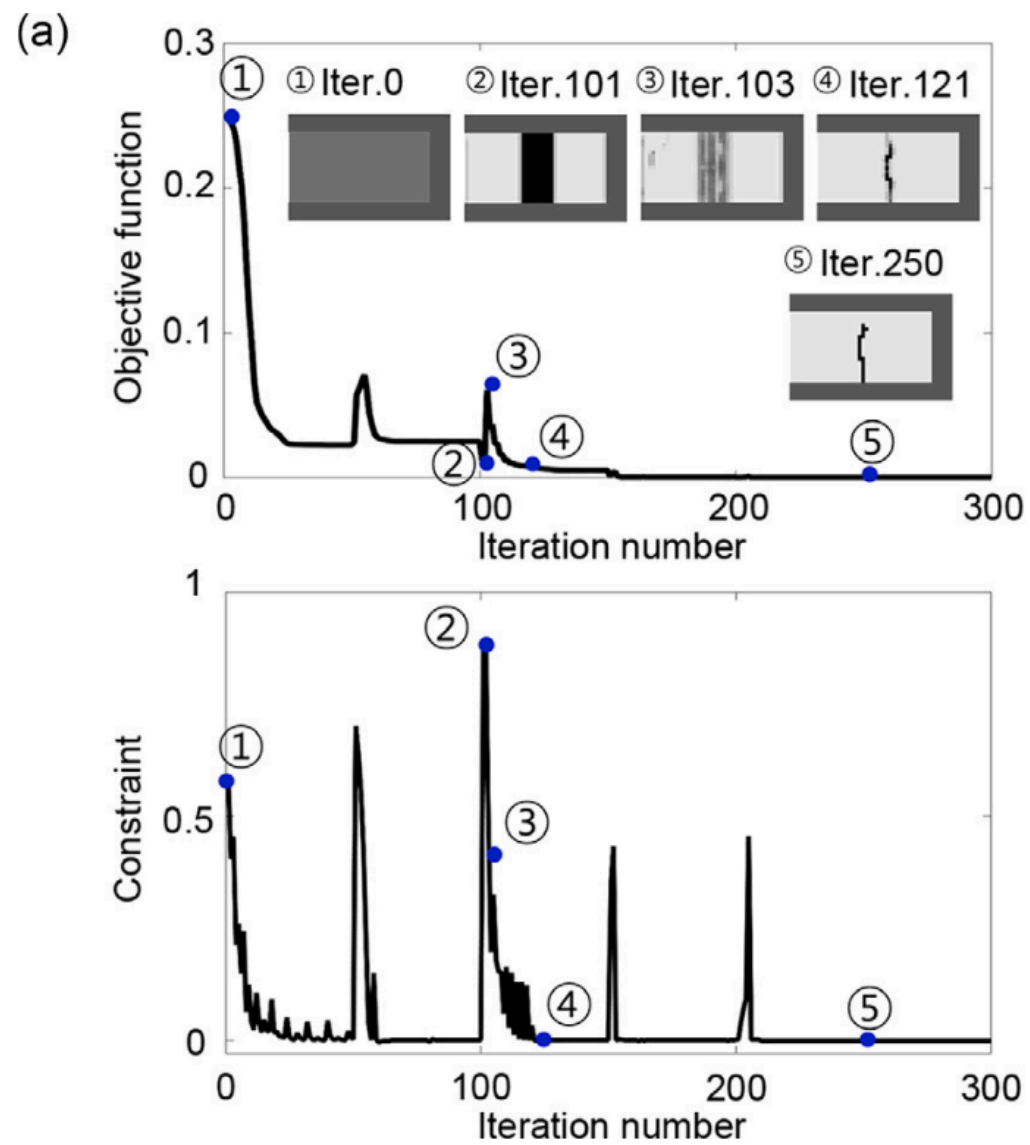
$$\frac{\partial p}{\partial \chi_e} = [\mathbf{K} - \omega^2 \mathbf{M}]^{-1} \cdot \frac{\partial [\mathbf{K} - \omega^2 \mathbf{M}]}{\partial \chi_e} \cdot \mathbf{P}. \quad (14)$$

(b)



(c)





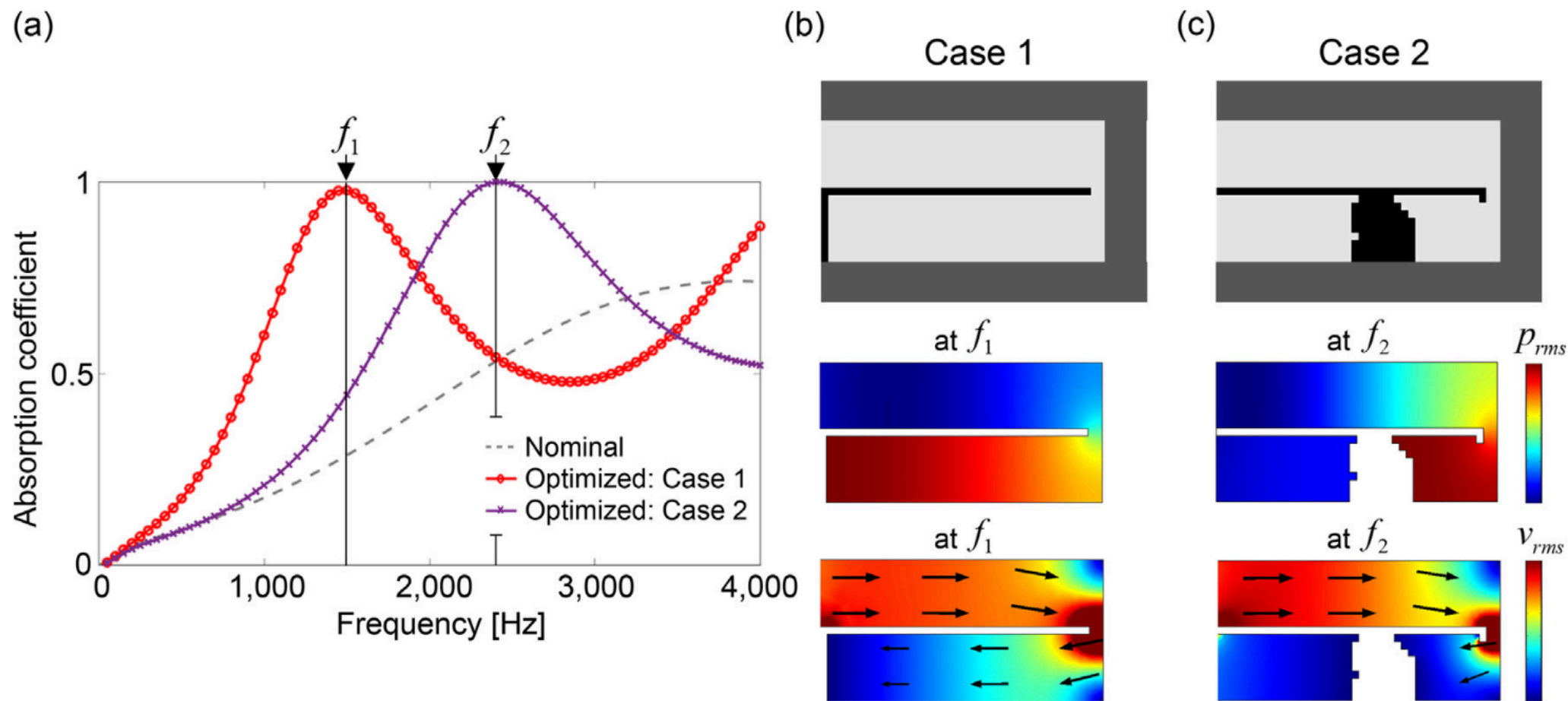


Fig. 4. Topology optimization results at single target frequencies $f_1 = 1500$ Hz (Case 1) and $f_2 = 2400$ Hz (Case2) under the restricted quarter-wavelength resonance

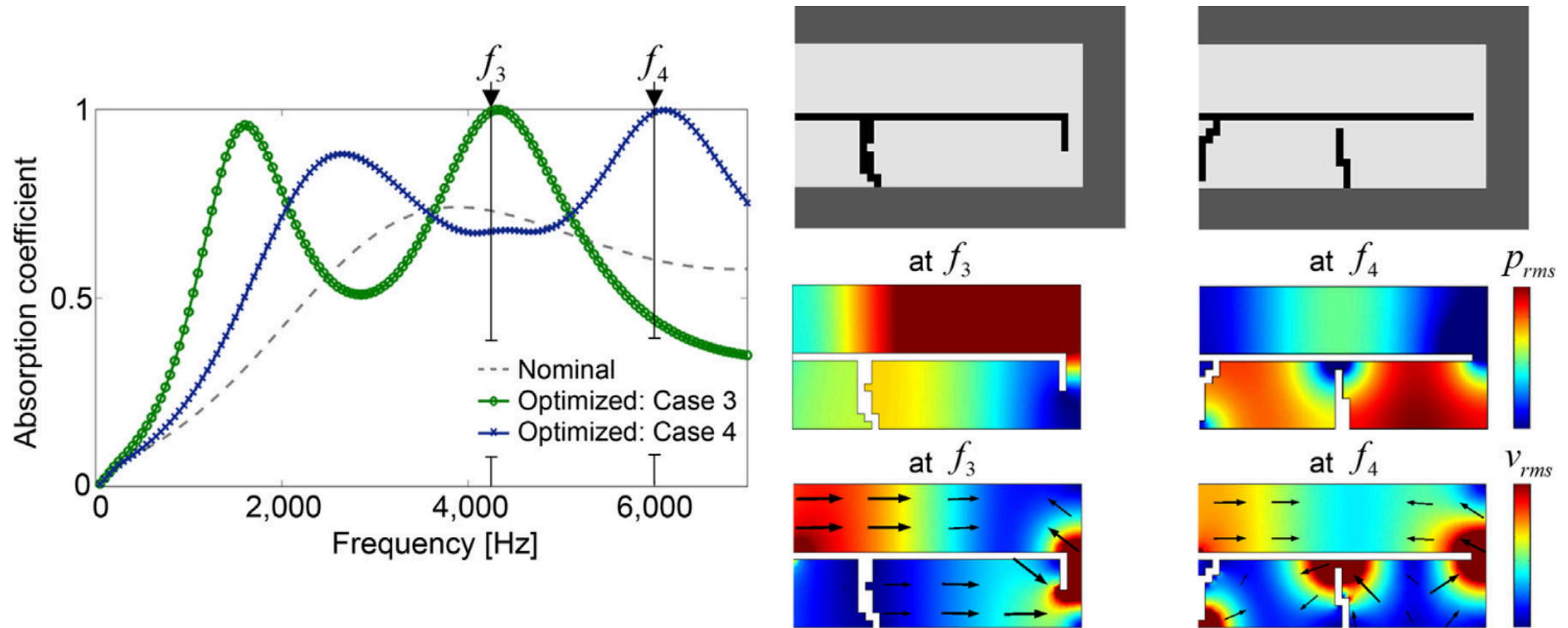


Fig. 5. Topology optimization results at single target frequencies $f_3 = 4200$ Hz (Case 3) and $f_4 = 6000$ Hz (Case 4) under the restricted a three-quarter-wavelength resonance forms to achieve the sound absorption coefficient nearly equal to 1 at $f = f_3$, while the quarter-wavelength resonance makes the first absorption peak at the frequency lower than 2000 Hz.

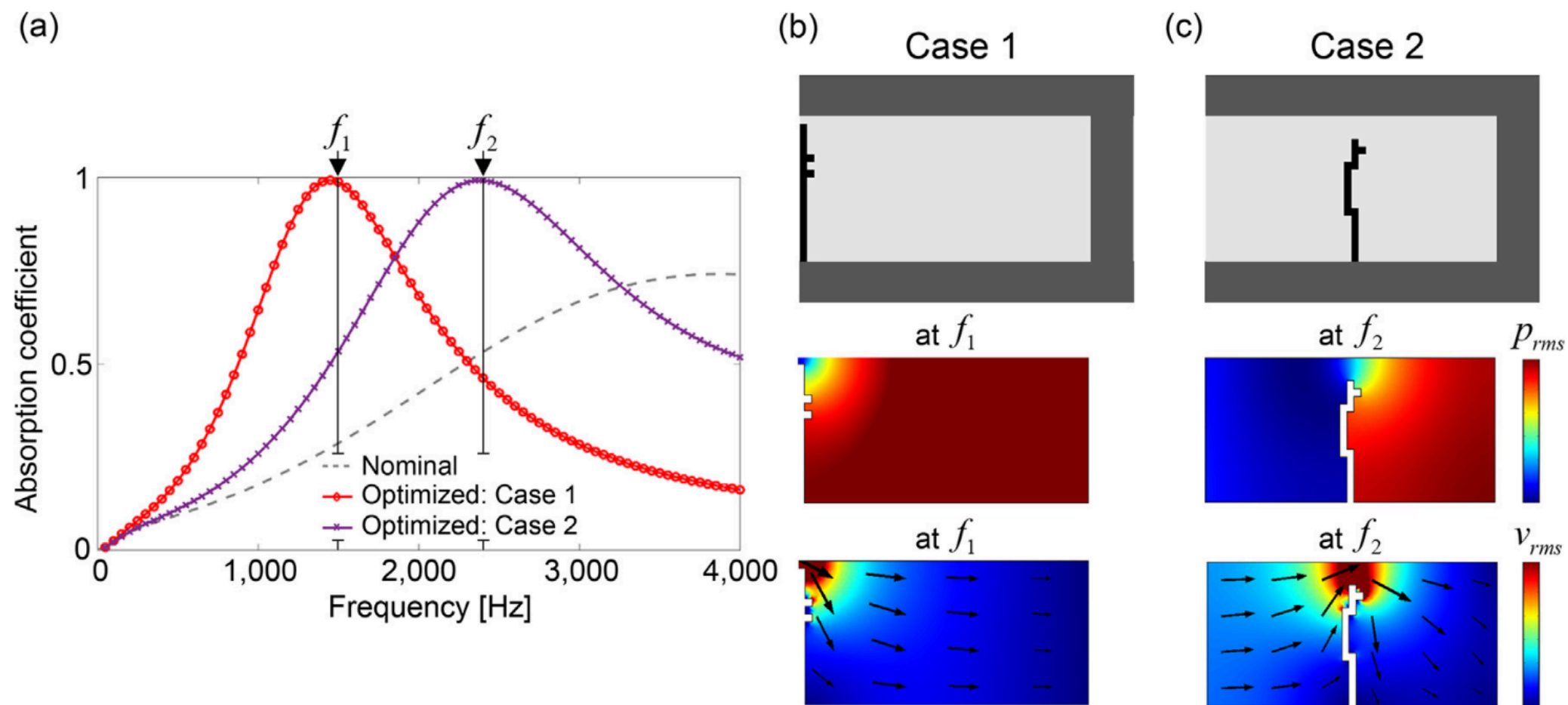


Fig. 6. Topology optimization results at single target frequencies $f_1 = 1500$ Hz (Case 1) and $f_2 = 2400$ Hz (Case2) under the full design

Helmholtz resonance

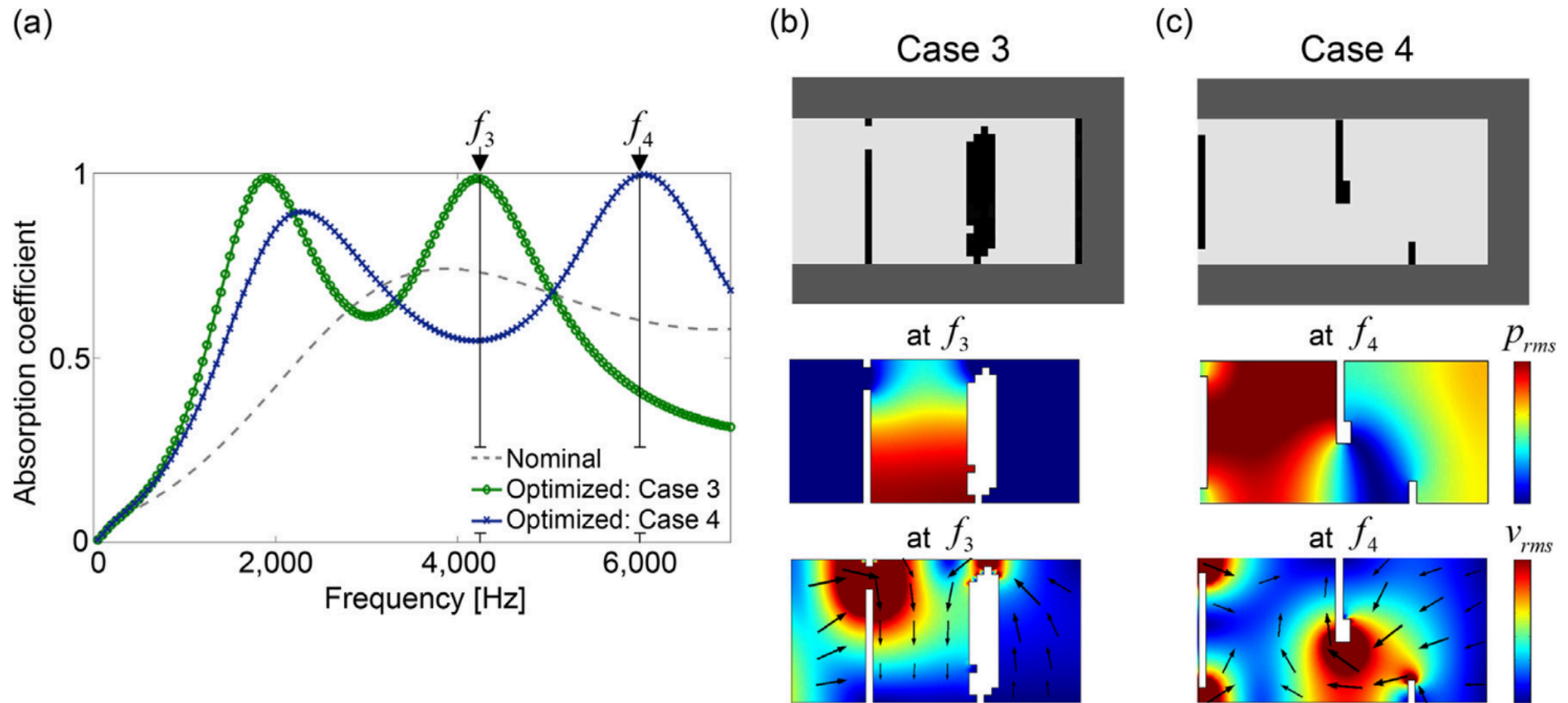
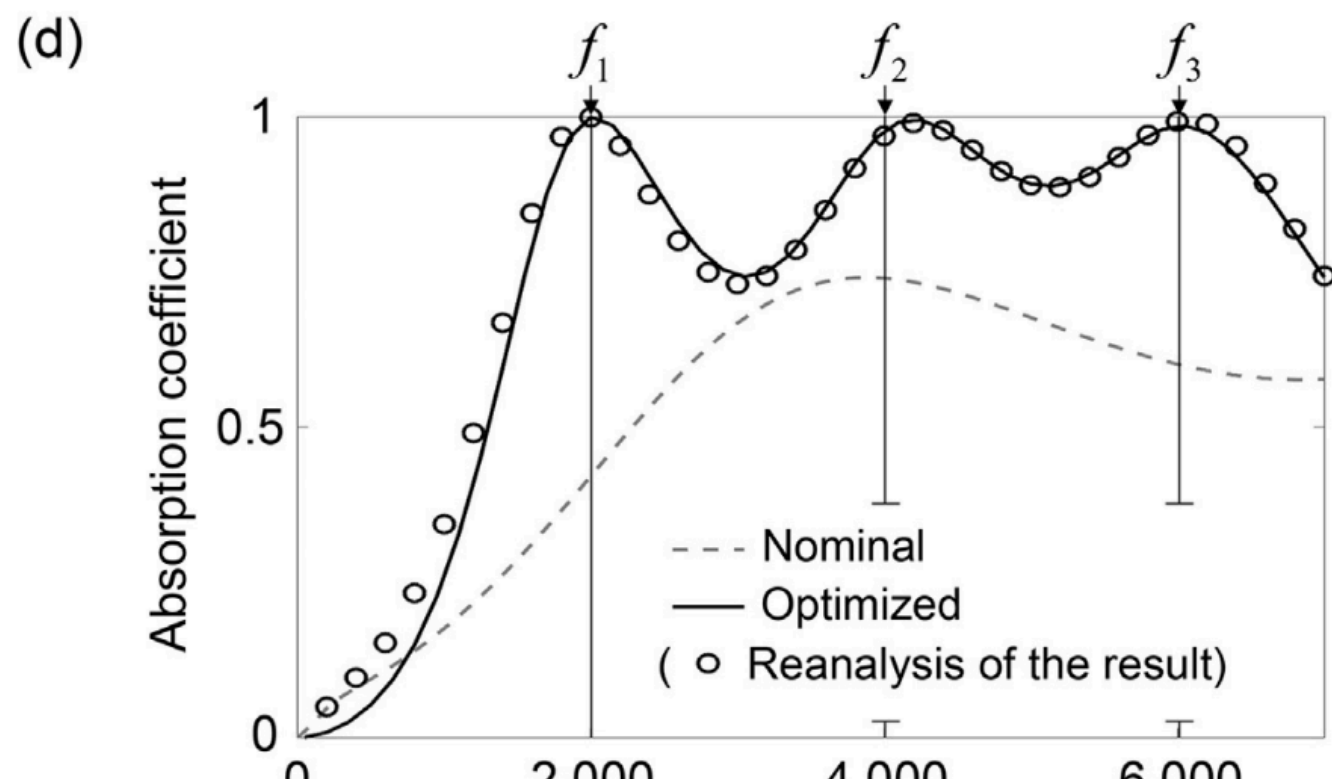
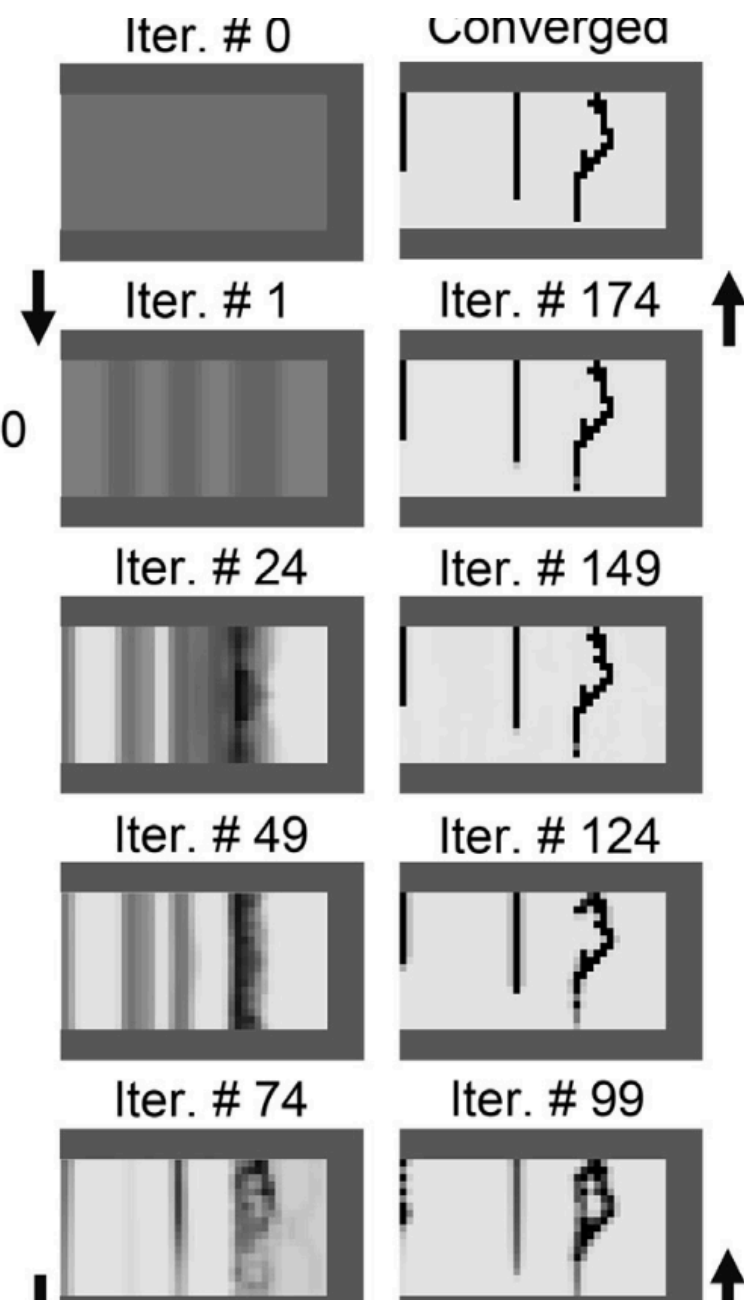
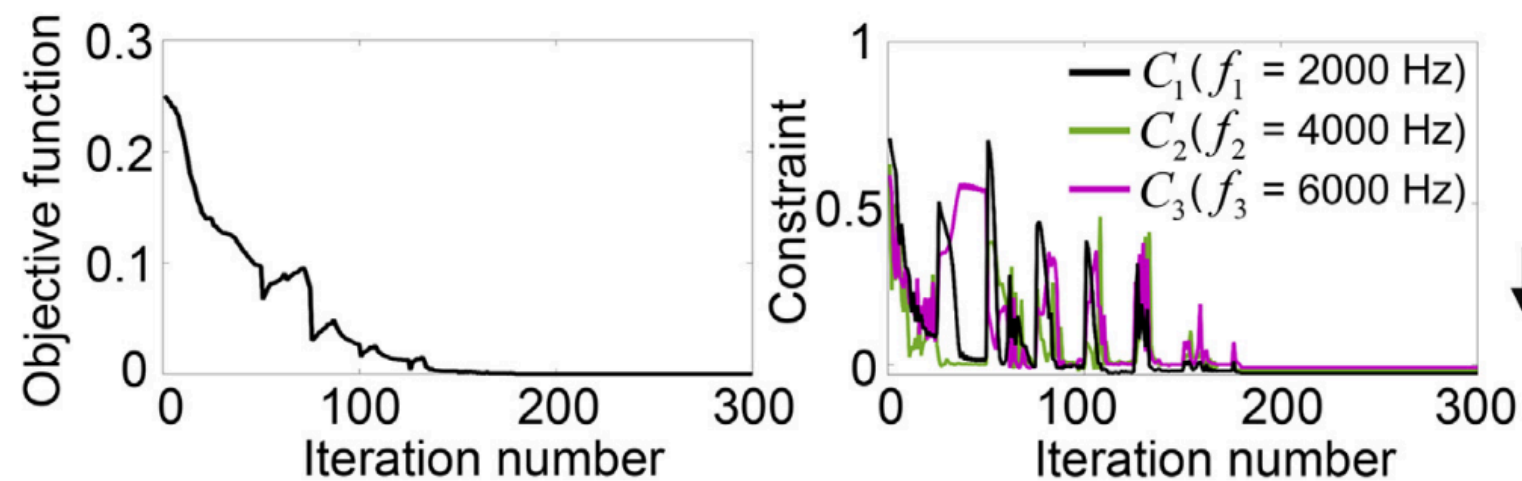
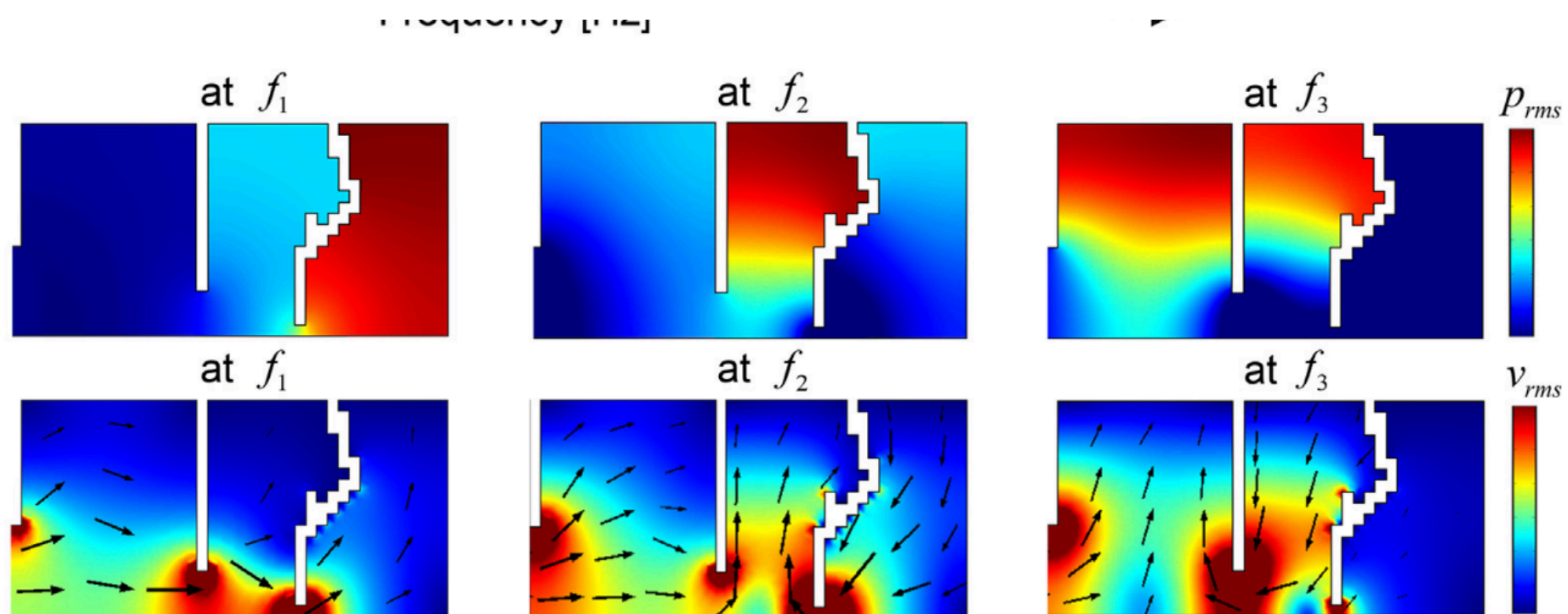


Fig. 7. Topology optimization results at single target frequencies $f_3 = 4200$ Hz (Case 3) and $f_4 = 6000$ Hz (Case 4) under the full design



(e)

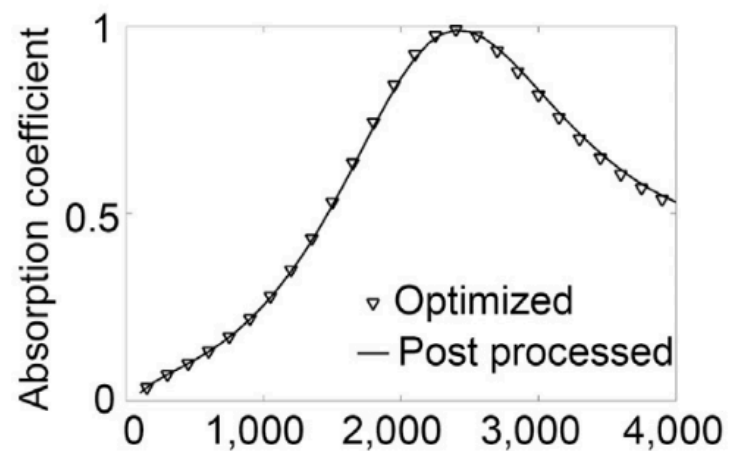
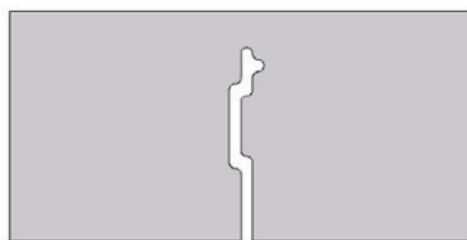


(a)

Optimized



Post processed

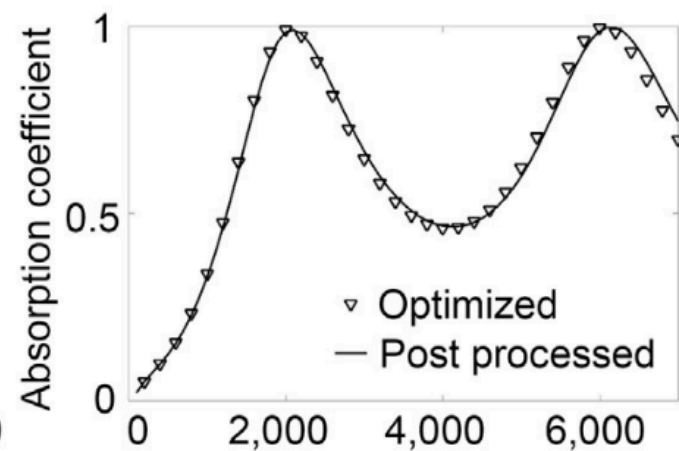
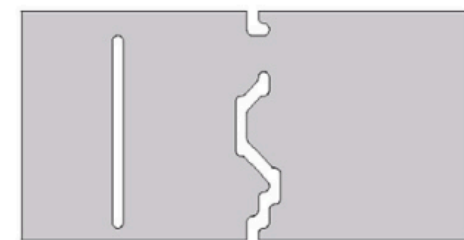


(b)

Optimized



Post processed



(c)

Optimized



Post processed

