

Lecture 5: (Non-Linear Control)

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5.1 Announcements

- EECS 298 is a companion class that we can audit if people are looking for more advanced lectures:
 - Just sitting in might be beneficial
- The schedule is going to lose a week to account for spring break
- List of papers for lab reports will be up Wednesday or Thursday next week
 - Lab groups will each choose 2 papers for presentations
 - In groups of 2
- HW2 will be available soon

5.2 Model-based control

PD and PID control laws can be applied to real systems to control them

- $m\ddot{x}(t) - b\dot{x}(t) + Kx(t) = u(t)$ where $u(t)$ is a force or input to the system
- $u(t) = m(\ddot{x}^{dest}(t) - K_d\dot{e}(t) - K_p e(t)) + b\dot{x}(t) + kx(t)$
 where $(\ddot{x}^{dest}(t) - K_d\dot{e}(t) - K_p e(t))$ is called the feed-forward term, K_d is the derivative coefficient while K_p is the proportional coefficient
 - Servo-Based components
 - * the \ddot{x}^{dest} , K_d , and K_p elements are independent of the model
 - * Tune PD or PID feedback portions to drive error to 0
 - Model Based components
 - * the m , b , and k elements are part of the system
 - * These elements cancel system dynamics

5.2.0.1 Model-based Control law using approximations

We can estimate values of m, b, k with $\hat{m}, \hat{b}, \hat{k}$ to obtain:

$$u(t) = \hat{m}(\ddot{x}^{dest}(t) - K_d\dot{e}(t) - K_p e(t)) + \hat{b}\dot{x}(t) + \hat{k}x(t)$$

Substituting this into the original equation:

$$\ddot{e} + K_d \dot{e} + K_p e = (1 - \frac{m}{\hat{m}})\ddot{x} + (\frac{\hat{b} - b}{\hat{m}})\dot{x} + (\frac{\hat{k} - k}{\hat{m}})x$$

Advantages:

- We can decompose the control law into model-dependent and model-independent
- This lets us use the model independent part in any system
- Great for learning algorithms

Disadvantages:

- If model parameters have errors then error will not go to 0
- We estimate m, b, k and have to tune them with trial and error

5.3 Fully Actuated vs Under-actuated

A control system with coordinates q ($q \in \mathbb{R}^6$ for Quadrotors covered) and inputs u is fully actuated if it can achieve any instantaneous acceleration in q .

A necessary condition is for the number of control inputs to be at least as great as the number of degrees of freedom.

Under-actuated Systems are:

- Insufficient number of inputs
- Structure of dynamics
- actuator limits

For “control-affine” systems, simple necessary and sufficient conditions for being fully actuated.

$$\ddot{q} = f(q, \dot{q}) + g(q, \dot{q})u$$

Needs rank of $g(q, \dot{q}) = \dim(q)$

5.4 Holonomic and nonholonomic

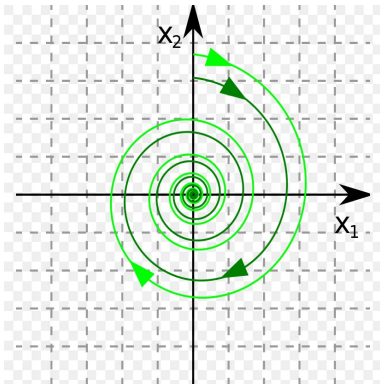
Given a dynamical system with coordinates q

- Holonomic constraints are constraints on the configuration q
- Nonholonomic constraints include constraints on the velocities \dot{q} which can not be integrated into holonomic constraints.

Example:

A toy car driving on the ground can achieve any configuration $q = (x, y, \phi)$ but it cannot drive sideways. This constraint is non-holonomic because it is on the velocity not the configuration

5.5 Phase Portraits



Specific vector field in 2D

Generated by plotting the $f(x)$ over the domain and placing the resulting vectors

- Circles are for equilibrium points (0 vectors)
- Full circles are stable
- Empty circles are unstable
- Follow vectors to equilibrium points

5.6 Lyapunov Stability theorem

For a system:

$$\dot{x} = f(x) | x \in R^n, f : R^n \rightarrow R^n$$

The equilibrium point $x=0$ is stable in $D \subset R^n$ iff there exists a smooth function $V : D \subset R^n \rightarrow R$ such that

$$V(0) = 0$$

$$V > 0 \forall x \in D - 0$$

$$\dot{V} \leq 0 \forall x \in D$$

Lie Derivatives:

For system $\dot{x} = f(x)$

function $V(x)$

The Lie derivative of a function $V(x)$ along a vector field f describes how the function changes along solutions of the differential equation

$$\frac{d}{dt} V(x(t)) = \mathcal{L}_f V(x(t))$$

$$\mathcal{L}_f V(x) = \frac{dV}{dx}(x) * f(x)$$

Using this notation, Lyapunov's stability theorem requires $\mathcal{L}_f V(x) < 0$

5.7 Input output linearization

Also known as partial feedback linearization

Used to convert a nonlinear system into an equivalent linear system

- State equations: $\dot{x} = f(x) + g(x)u$
- output: $y=h(x)$
- goal is to design $u = \alpha(x) + \beta(x)v$
- such that $\dot{y} = v$
- use rate of change of output: $\dot{y} = \mathcal{L}_f h + (\mathcal{L}_g h)u$
- if $\mathcal{L}_g h \neq 0$ then $u = \frac{1}{\mathcal{L}_g h}(-\mathcal{L}_f h + \dot{y}^{des} + k(y^{des} - y))$
- where v is virtual and $v = \dot{y}^{des} + k(y^{des} - y)$
- plugging into our state equations gives $\dot{y} - \dot{y}^{des} + k(y^{des} - y) = \dot{y} = v$
- if $\mathcal{L}_g h = 0$, then the rate of change of output is independent to u and we need to use a higher derivative

Higher Derivatives:

- Relative degree r : index of first nonzero term in the sequence
- r is the first nonzero term of $\mathcal{L}_g \mathcal{L}_f^{r-1} h$
- $u = \frac{1}{\mathcal{L}_g \mathcal{L}_f^{r-1} h}(-\mathcal{L}_f^r h + y_{dest}^{(r)} + k_1(y_{dest}^{(r-1)} - y^{(r-1)}) + \dots + k_r(y_{dest} - y))$

5.8 Multiple Input Multiple Output (MIMO) Systems

For a MIMO system, we have the state $x \in R^n$ and input $u \in R^m$. The state equation can be written in the form

$$\dot{x} = f(x) + g(x)u$$

$$y = h(x)$$

For this system, we assume that the output has a relative degree r . With this we can apply the Nonlinear feedback law

$$u = (\mathcal{L}_g \mathcal{L}_f^{r-1} h)^{-1}(-\mathcal{L}_f^r h + v)$$

Which gets us to the equivalent system

$$y^{(r)} = v$$

For a fully actuated robotic arm (n joints, n actuators):

$$M(q)\ddot{q} + C(q, \dot{q})\dot{q} + N(q) = \tau$$

where:

- M is the positive definite $n \times n$ inertia matrix
- C is the $n \times n$ matrix of Coriolis and centripetal forces
- N is the n -dimensional vector of gravitational forces
- τ is the n -dimensional vector of actuator forces and torques

$$M(q)\ddot{q} + C(q, \dot{q})\dot{q} + N(q) = \tau$$

$$x = \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \begin{bmatrix} q \\ \dot{q} \end{bmatrix} \quad u = \tau \in \mathbb{R}^n$$

$$\dot{x} = \begin{bmatrix} x_2 \\ -M(x_1)^{-1}(N(x_1) + C(x_1, x_2)x_2) \end{bmatrix} + \begin{bmatrix} 0 \\ M(x_1)^{-1} \end{bmatrix} u$$

$$h(x) = x_1$$

$$f(x) = \begin{bmatrix} x_2 \\ -M(x_1)^{-1}(N(x_1) + C(x_1, x_2)x_2) \end{bmatrix} \quad g(x) = \begin{bmatrix} 0 \\ M(x_1)^{-1} \end{bmatrix}$$

$$h(x) = x_1$$

$$\mathcal{L}_g h = 0, \quad \mathcal{L}_g \mathcal{L}_f h \neq 0$$

$$u = (\mathcal{L}_g \mathcal{L}_f h)^{-1} ((-\mathcal{L}_f^2 h + \ddot{y}^{des} + k_1(\dot{y}^{des} - \dot{y}) + k_2(y^{des} - y))$$

With the control law

$$u = M(x_1)(M(x_1)^{-1}(N(x_1) + C(x_1, x_2)x_2) + \ddot{y}^{des} + k_1(\dot{y}^{des} - \dot{y}) + k_2(y^{des} - y))$$

5.9 image sources

- Phase Diagram: https://favpng.com/png_view/mathematics-equilibrium-point-phase-portrait-mathematics-differential-equation-chaos-theory-png/siNtyytG