

CT Exam 06/07/2023

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Part A

```
%set up
format("short","g")
clear
clc
```

```
%defining constants:
km=5.7;
mF=0.8;
mM=1e-5;
mL=0.3;
dF=6;
kF=6000;
db=1;
kb=800;
d=3;
```

TASK1

a)

```
%defining state matrices:
A=[-(db+d)/mL 0 d/mL 0 db/mL kb/mL;
    1 0 0 0 0 0;
    d/mF 0 -(dF+d)/mF kF/mF 0 0;
    0 0 -1 0 0 0;
    db/mM 0 0 0 -db/mM -kb/mM;
    -1 0 0 1 0 0]
```

```
A = 6x6
   -13.333         0         10         0         3.3333        2666.7
         1         0         0         0         0         0
         3.75         0        -11.25        7500         0         0
         0         0         -1         0         0         0
        1e+05         0         0         0        -1e+05        -8e+07
        -1         0         0         1         0         0
```

```
B=[0; 0; -km/mF; 0; km/mM; 0]
```

```
B = 6x1
         0
         0
        -7.125
         0
         0
         0
```

```

0
5.7e+05
0

```

```

C=[0 1 0 0 0 0;
    0 0 0 1 0 0;
    0 0 0 0 0 1]

```

```

C = 3x6
    0     1     0     0     0     0
    0     0     0     1     0     0
    0     0     0     0     0     1

```

```

D=[0; 0; 0]

```

```

D = 3x1
    0
    0
    0

```

b)

```

%to get tf for out xload
[num1, den1]=ss2tf(A,B,C(1,:),D(1,:));
sys1=tf(num1, den1)

```

```

sys1 =

          1.9e06 s^3 + 1.425e07 s^2 + 1.425e10 s + 0.0002255
-----
s^6 + 1e05 s^5 + 2.135e06 s^4 + 7.576e08 s^3 + 7.52e09 s^2 + 1.099 s

```

Continuous-time transfer function.

```

%to get tf for out xframe
[num2, den2]=ss2tf(A,B,C(2,:),D(2,:));
sys2=tf(num2, den2)

```

```

sys2 =

      7.125 s^4 + 7.126e05 s^3 + 1.9e04 s^2 - 1.393e-06 s - 2.384e-24
-----
s^6 + 1e05 s^5 + 2.135e06 s^4 + 7.576e08 s^3 + 7.52e09 s^2 + 1.099 s

```

Continuous-time transfer function.

```

%to get tf for out xmotor
[num3, den3]=ss2tf(A,B,C(3,:),D(3,:));
sys3=tf(num3, den3)

```

```

sys3 =

      -1.9e06 s^3 - 1.354e07 s^2 - 1.425e10 s + 0.0005461
-----
s^6 + 1e05 s^5 + 2.135e06 s^4 + 7.576e08 s^3 + 7.52e09 s^2 + 1.099 s

```

Continuous-time transfer function.

c)

```
%getting poles of each system
```

```
P1=pole(sys1)
```

```
P1 = 6x1 complex
      0 +      0i
     -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
   -1.4611e-10 +      0i
```

```
P2=pole(sys2)
```

```
P2 = 6x1 complex
      0 +      0i
     -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
   -1.4611e-10 +      0i
```

```
P3=pole(sys3)
```

```
P3 = 6x1 complex
      0 +      0i
     -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
   -1.4611e-10 +      0i
```

```
%getting eigenvalues for each system
```

```
e1=eig(sys1)
```

```
e1 = 6x1 complex
      0 +      0i
     -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
   -1.4611e-10 +      0i
```

```
e2=eig(sys2)
```

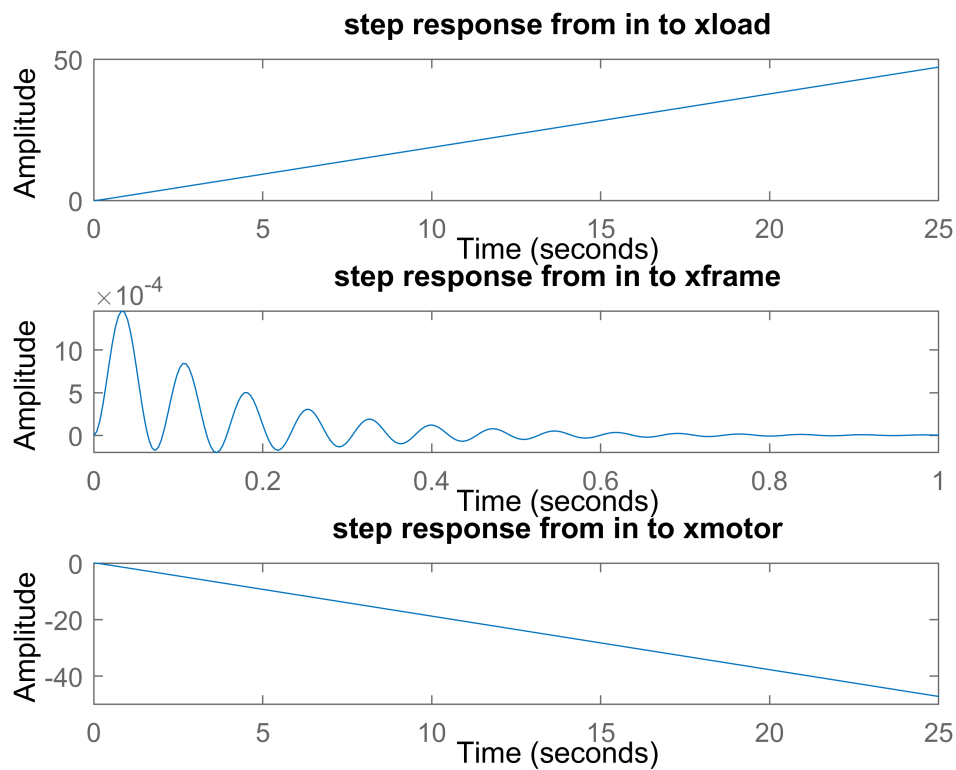
```
e2 = 6x1 complex
      0 +      0i
     -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
   -1.4611e-10 +      0i
```

```
e3=eig(sys3)
```

```
e3 = 6x1 complex
      0 +      0i
    -1e+05 +      0i
    -5.5998 +    86.204i
    -5.5998 -    86.204i
    -10.077 +      0i
    -1.4611e-10 +      0i
```

d)

```
%plotting step responses for each system
figure()
subplot(3,1,1)
step(sys1)
title("step response from in to xload")
subplot(3,1,2)
step(sys2)
title("step response from in to xframe")
subplot(3,1,3)
step(sys3)
title("step response from in to xmotor")
```



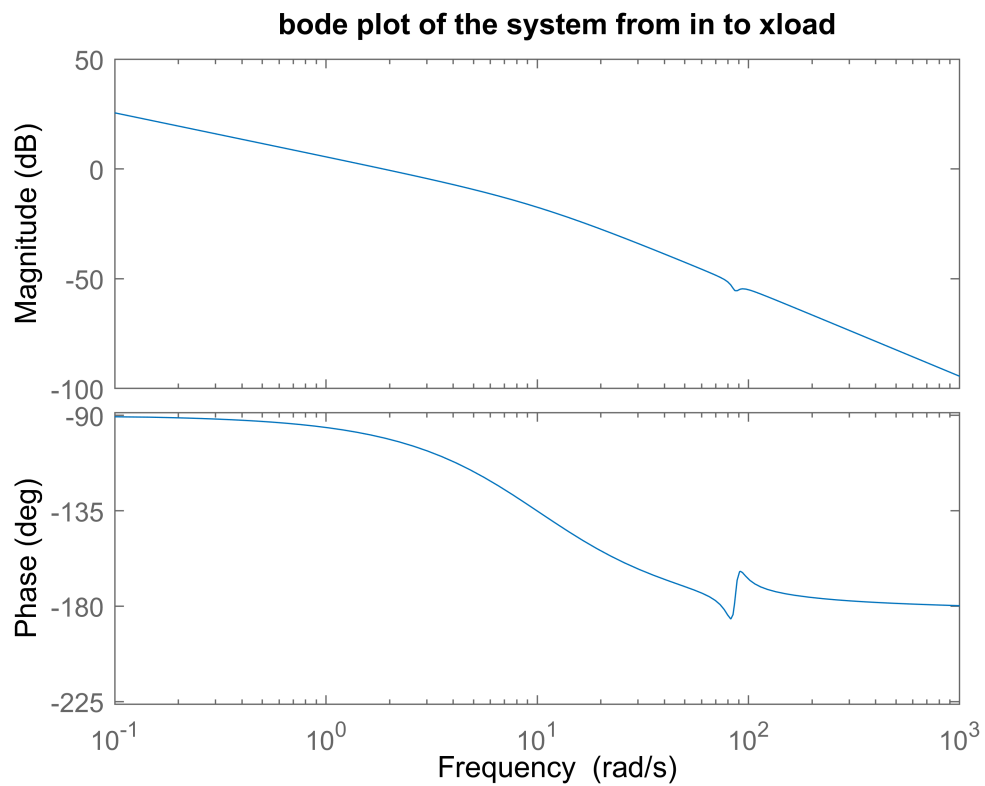
Step response from in to xload implies unstable behaviour (goes to infinity).

Step response from in to xframe behaves in a damped oscillatory manner (stable).

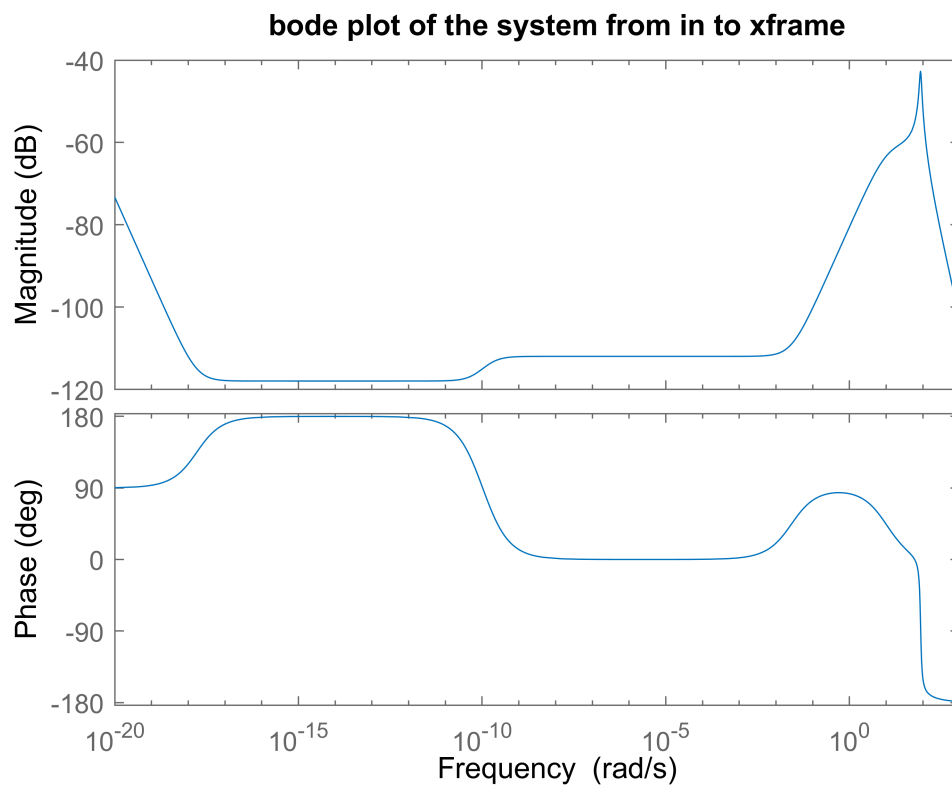
Step response from in to xmotor shows instability (goes to minus infinity).

e)

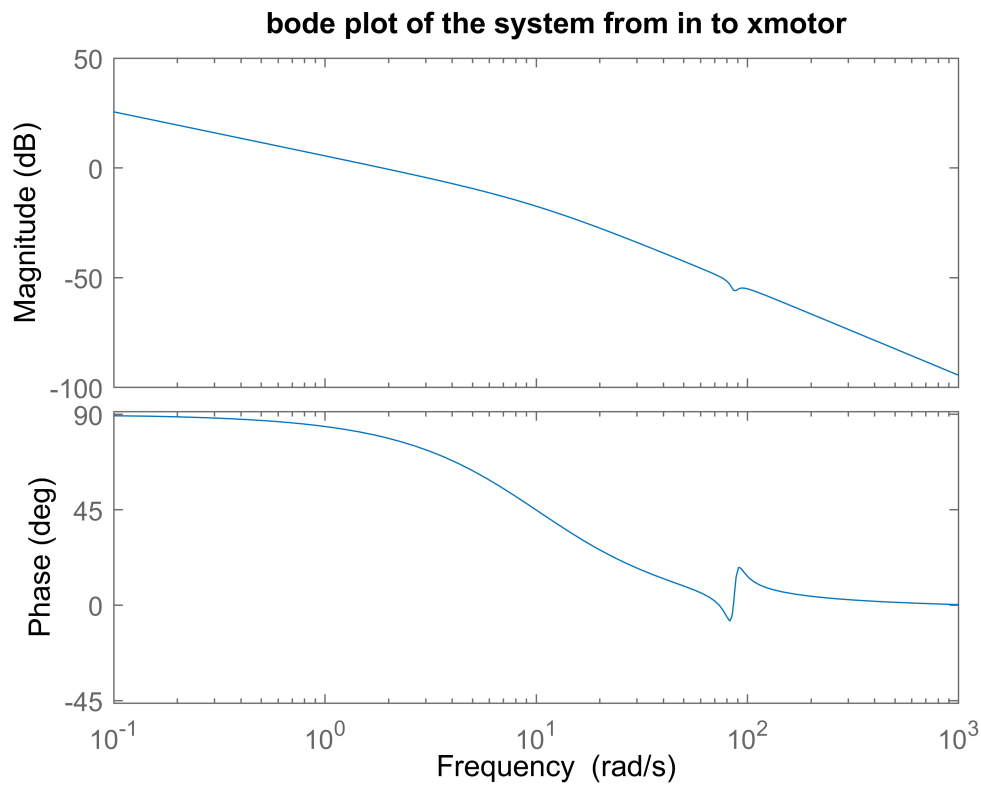
```
%plotting bode plots for each system  
figure()  
bode(sys1)  
title("bode plot of the system from in to xload")
```



```
figure()  
bode(sys2)  
title("bode plot of the system from in to xframe")
```



```
figure()  
bode(sys3)  
title("bode plot of the system from in to xmotor")
```



TASK2

a)

```
%checking controllability of the system
Cc=ctrb(A,B);
rank_of_Cc=rank(Cc)
```

```
rank_of_Cc =
    1
```

```
controllability=length(A)-rank_of_Cc
```

```
controllability =
    5
```

Given that the rank of the controllability matrix of the system is 1 while the order of the system is 6, the system has controllability deficiency of 5, therefore it is uncontrollable.

```
%checking observability for each output
%load
O1 = obsv(A, C(1,:));
rank_of_O_load = rank(O1)
```

```
rank_of_0_load =  
3
```

```
%xframe  
O2 = obsv(A, C(2,:));  
rank_of_0_frame = rank(O2)
```

```
rank_of_0_frame =  
3
```

```
%xmotor  
O3 = obsv(A, C(3,:));  
rank_of_0_motor = rank(O3)
```

```
rank_of_0_motor =  
3
```

The system is unobservable from the input to any of the outputs used, individual or combined. Beacouse the rank of the observability matrix for each of them is 3.

TASK3

```
%defining system  
G6l=tf(num1, den1)
```

```
G6l =  
  
1.9e06 s^3 + 1.425e07 s^2 + 1.425e10 s + 0.0002255  
-----  
s^6 + 1e05 s^5 + 2.135e06 s^4 + 7.576e08 s^3 + 7.52e09 s^2 + 1.099 s  
  
Continuous-time transfer function.
```

a)

```
%poles  
P=pole(G6l)
```

```
P = 6x1 complex  
0 + 0i  
-1e+05 + 0i  
-5.5998 + 86.204i  
-5.5998 - 86.204i  
-10.077 + 0i  
-1.4611e-10 + 0i
```

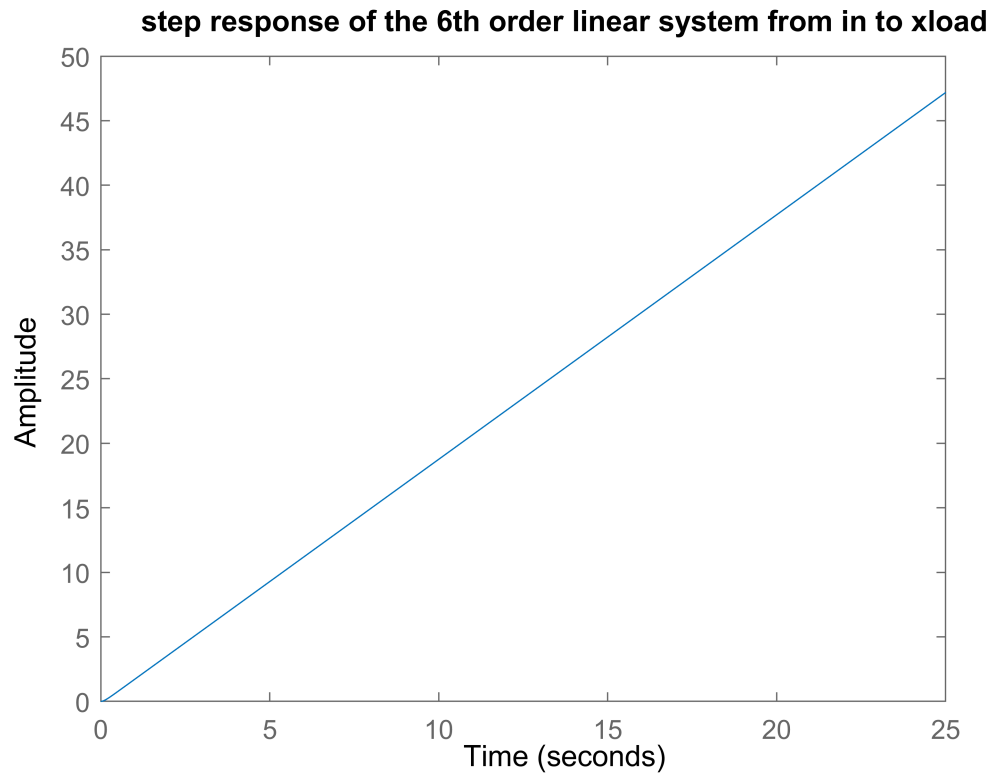
```
%zeros  
Z=zero(G6l)
```

```
Z = 3x1 complex  
-3.7501 + 86.523i  
-3.7501 - 86.523i  
-1.5827e-14 + 0i
```

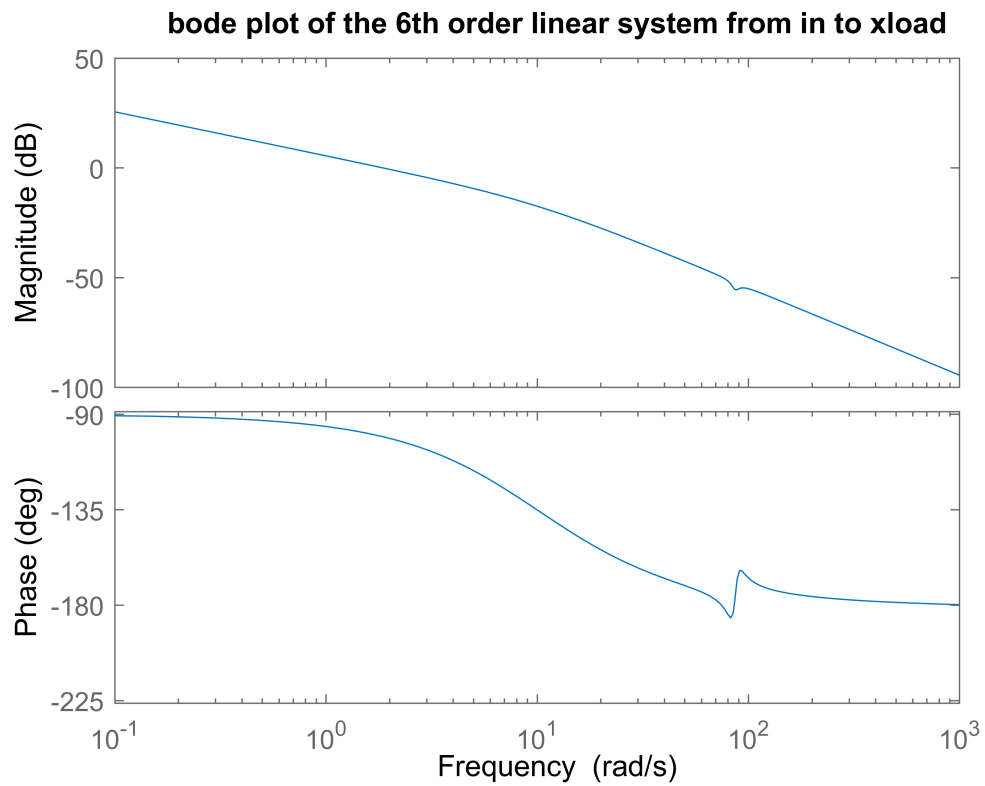

Based on the poles of the 6th order linear system, it is marginally stable since all of the real values of the complex poles are in the left hand plane ie. negative except for one at 0.

b)

```
%plotting step and bode  
figure()  
step(G6l)  
title("step response of the 6th order linear system from in to xload")
```



```
figure()  
bode(G6l)  
title("bode plot of the 6th order linear system from in to xload")
```

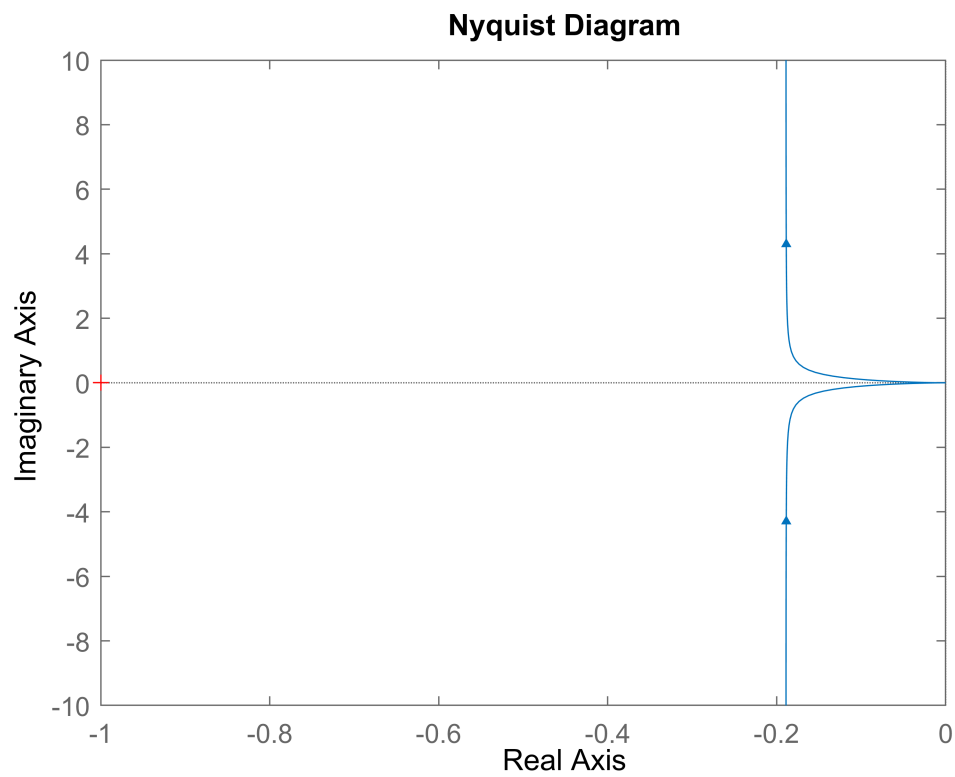


Step response of this system shows instability (goes to infinity)

From Bode plot we conclude that the open loop system is unstable due to unsatisfied phase margin criterion.

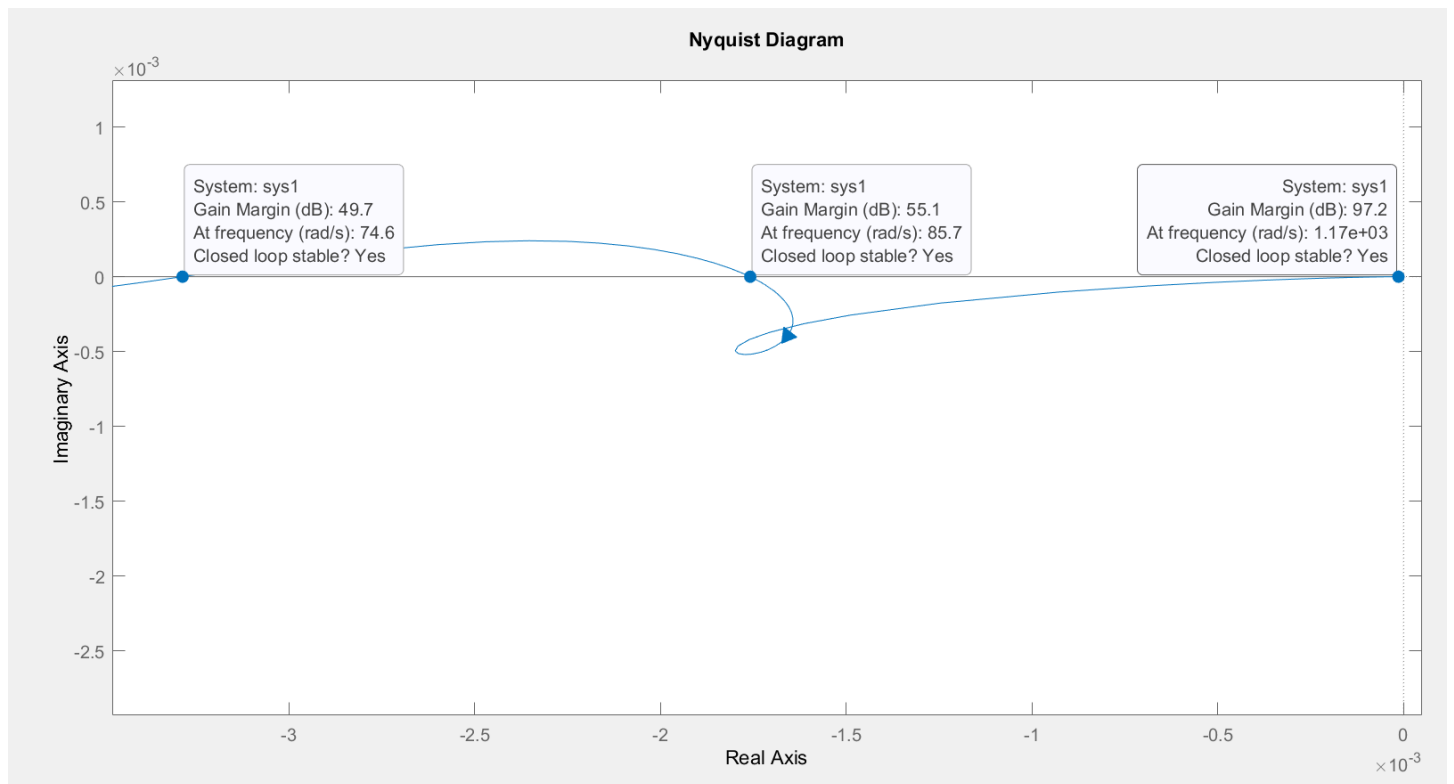
c)

```
%plotting nyquist and rlocus  
figure()  
nyquist(G61)
```



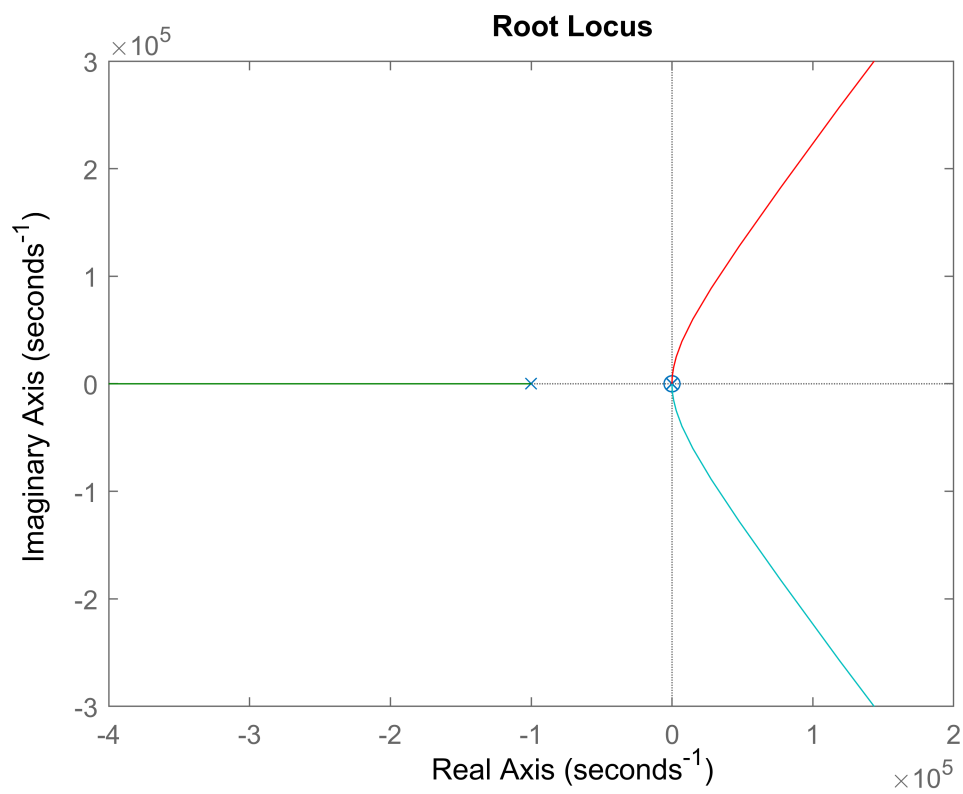
Nyquist plot shows that the plot crosses unit circle at frequency of 1.86 rad/s, giving phase margin 79.5 deg.

For phase = $-\pi$ it has 3 different gains shown on the plot below:

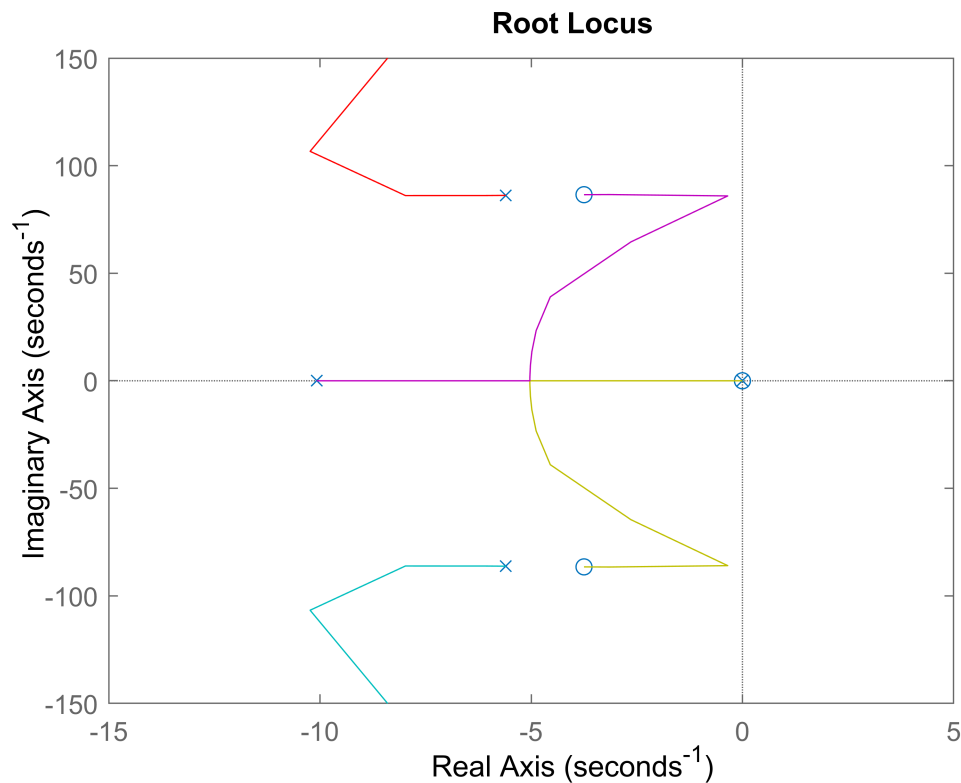


therefore the open loop system should be unstable.

```
figure()  
rlocus(G61)
```



```
figure()  
rlocus(G61)  
xlim([-15 5])  
ylim([-150 150])
```



locus plot shows the existence of 5 poles and 3 zeros where one pole is insignificant due to its distance from the unit circle, while the other pole gives the system marginal stability due to its location at point (0,0). We have 3 significant poles around the unit circle to the left of the imaginary axis.

Among the 3 zeros of the system one lies on the (0,0) point, while the other 2 are around the unit circle to the left of the imaginary axis.

This once again implies that the system is marginally stable.

d)

```
%checking if variable state feedback can be designed
[Abar,Bbar,Cbar,T,k]=ctrbf(A,B,C(1,:))
```

```
Abar = 6x6
    1.8605e-13    -8.6726e-13    -1.3417e-14    5.8057e-14    -4.9024e-16    3.2732e-16
    0.0015827     7.0951     2.1719    3.0011e-15    -1.212e-16    -6.884e-17
    2.3716     -3270.3     -768.84    415.22    1.4168e-15    1.8444e-16
    4.3069     -5944.3     -1397.3    755.18     2.8426    1.0347e-16
    2012      2025.6      2305.3    -1256.9    -18.017     3.5598
   -5.6569e+07    1.4563e+07   -4.7874e+07  2.6382e+07    93634     -1e+05
Bbar = 6x1
   -6.5888e-15
   -6.4752e-14
    3.0875e-11
    5.6133e-11
    1.7085e-10
    5.7e+05
Cbar = 1x6
```

```

    0.70711    0.183    -0.59834    0.3294    0    0
T = 6x6
   -3.5355e-05    0.70711   -9.4281e-05    0.00070711   -1.1785e-09    0.70711
   -0.00035459    0.183   -0.00095531   -0.96611   -1.1941e-08   -0.18204
    0.16921   -0.59834    0.45126   -0.22664    5.6407e-06    0.59863
    0.30764    0.3294    0.82042    0.12354    1.0255e-05   -0.32939
    0.93634    0    -0.3511    3.5114e-06   -4.3888e-06    0
    0    0   -1.25e-05    0    1    0
k = 1x6
    1    1    1    1    1    0

```

```

%displaying new
poles_new = eig(Abar)

```

```

poles_new = 6x1 complex
   -1e+05 + 0i
   -5.5998 + 86.204i
   -5.5998 - 86.204i
   -10.077 + 0i
   -1.6665e-15 + 0i
   -2.3269e-12 + 0i

```

```

%displaying the controllable and uncontrollable poles
controllable_poles = eig(Abar(2:6,2:6))

```

```

controllable_poles = 5x1 complex
   -1e+05 + 0i
   -5.5998 + 86.204i
   -5.5998 - 86.204i
   -10.077 + 0i
   -2.8532e-12 + 0i

```

```

uncontrollable_poles = eig(Abar(1,1))

```

```

uncontrollable_poles =
    1.8605e-13

```

$[Abar, Bbar, Cbar, T, k] = ctrbf(A, B, C)$ decomposes the state-space system represented by A, B, and C into the controllability staircase form, Abar, Bbar, and Cbar, described above. T is the similarity transformation matrix and k is a vector of length n , where n is the order of the system represented by A. Each entry of k represents the number of controllable states factored out during each step of the transformation matrix calculation. The number of nonzero elements in k indicates how many iterations were necessary to calculate T, and $\sum(k)$ is the number of states in Ac, the controllable portion of Abar.

The matrix K shows us that there are 5 controllable poles and 1 uncontrollable pole. This is confirmed by calculating eigenvalues of the Abar with subtracted 1st row and 1st column.

Matrix T is the transformation matrix such that $Abar = T^{-1}AT$.

The controllable system is unstable due to having a pole with positive real value.

The variable feedback controller can be designed to stabilize the system, because it has only one uncontrollable pole, which is very close to zero and is on the left side of the zero-pole plane (so it should be considered as equal to zero).

TASK4

a)

```
% minimal realization and pole-zero cancellation
%transfer function after performing minimal realization
G5m = minreal(G6l)
```

G5m =

$$\frac{1.9e06 s^2 + 1.425e07 s + 1.425e10}{s^5 + 1e05 s^4 + 2.135e06 s^3 + 7.576e08 s^2 + 7.52e09 s + 1.099}$$

Continuous-time transfer function.

```
%poles of the system after minimal realization
poles_min = pole(G5m)
```

```
poles_min = 5x1 complex
   -1e+05 + 0i
   -5.5998 + 86.204i
   -5.5998 - 86.204i
   -10.077 + 0i
   -1.4611e-10 + 0i
```

```
%zeros of the system after minimal realization
zeros_min = zero(G5m)
```

```
zeros_min = 2x1 complex
   -3.7501 + 86.523i
   -3.7501 - 86.523i
```

b)

We have decided to remove the pole which is the least significant due to its position with regards to the imaginary axis. The greater the distance from the unit circle the smaller is its impact on the dynamic behaviour of the system.

```
poles_new=[0 -10.077 -5.5998+86.204i -5.5998-86.204i]
```

```
poles_new = 1x4 complex
    0 + 0i    -10.077 + 0i ...
```

Zeros of the system do not have to be reduced.

```
zeros_new=zeros_min'
```

```
zeros_new = 1x2 complex
   -3.7501 - 86.523i    -3.7501 + 86.523i
```

After performing the reduction we have constructed new 4th order linear system shown below.

```
G4s=zpk(zeros_new,poles_new,1)
```

G4s =

$$\frac{(s^2 + 7.5s + 7500)}{s(s+10.08)(s^2 + 11.2s + 7462)}$$

Continuous-time zero/pole/gain model.

The result is displayed in the product form.

c)

```
%reconstructing the state matrices A, B, C, D:  
[A4, B4, C4, D4]=zp2ss(zeros_new,poles_new,1);  
A4
```

```
A4 = 4x4  
    -11.2    -86.386         0         0  
    86.386         0         0         0  
   -3.6993    0.43762   -10.077         0  
         0         0         1         0
```

B4

```
B4 = 4x1  
     1  
     0  
     1  
     0
```

C4

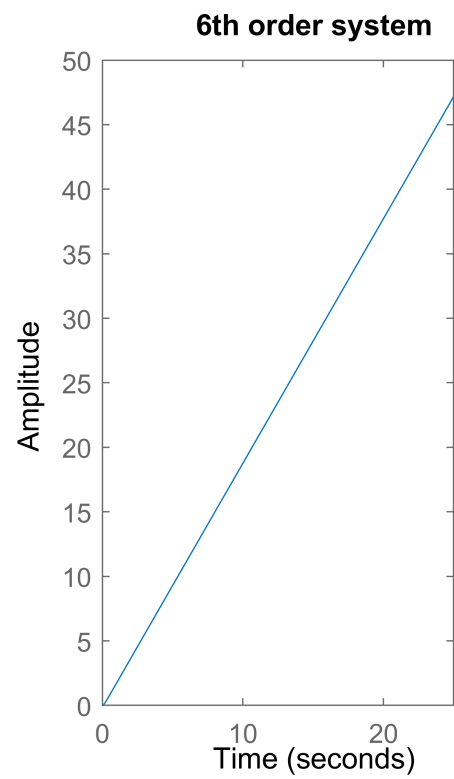
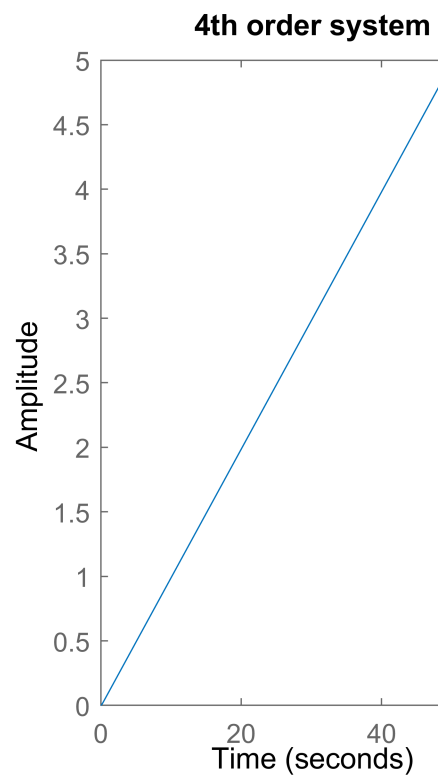
```
C4 = 1x4  
     0     0     0     1
```

D4

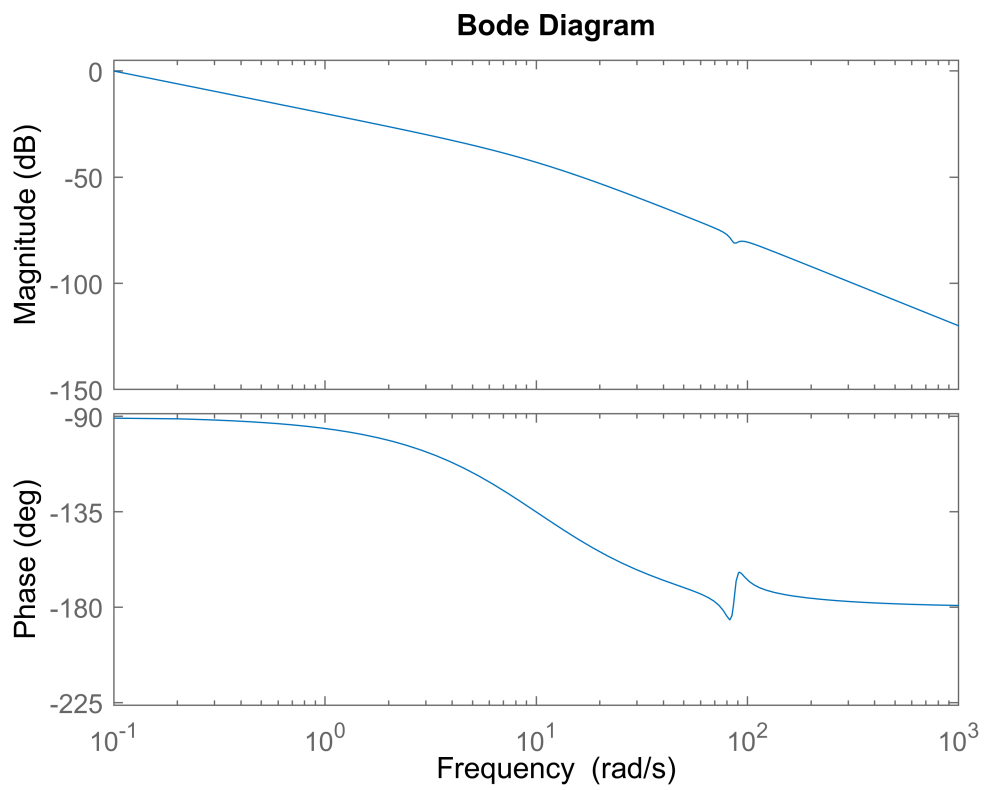
```
D4 =  
     0
```

d)

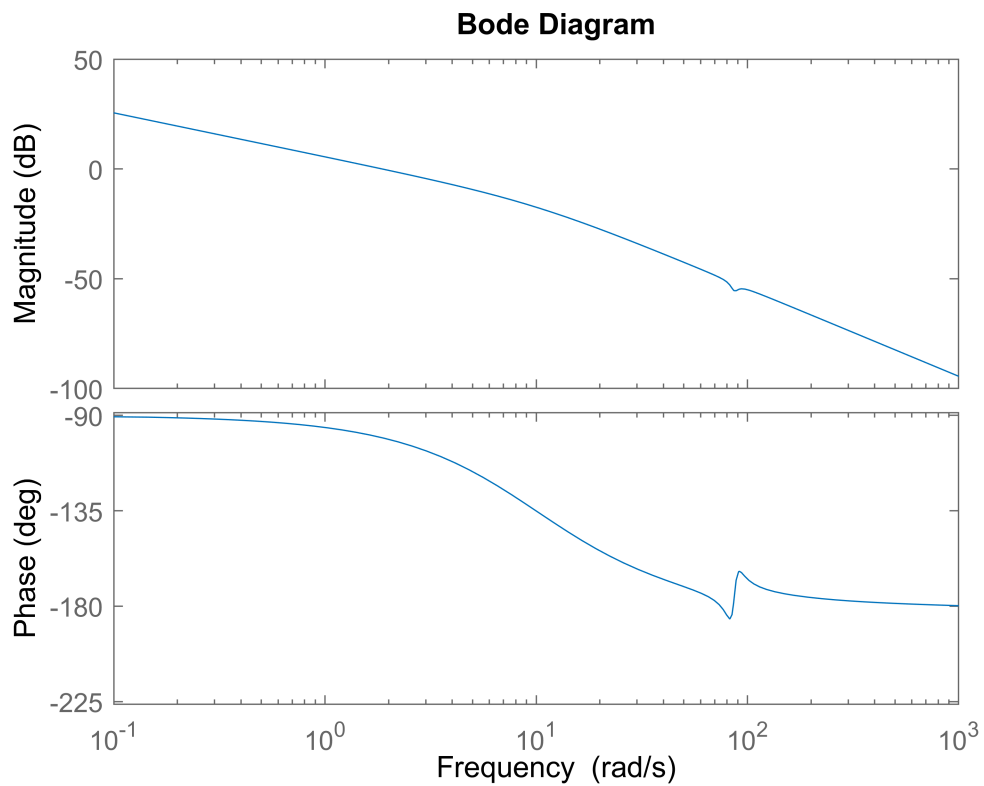
```
%plotting step response of the system  
figure()  
subplot(1,2,1)  
step(G4s)  
title("4th order system")  
subplot(1,2,2)  
step(G6l)  
title("6th order system")
```

```
%plotting bode plot of the system  
figure()  
bode(G4s)
```



```
figure()  
bode(G61)
```



As the original system was unstable, so is the reduced (4th order) one. The main difference is that the reduced system gain has decreased by a factor of ~ 10 . The dynamic behaviour of the reduced system is similar to the original one which shouldn't cause any losses. This is true for both the step and bode diagram.

TASK5

a)

```
% Check the controllability and observability of the system
system_order = length(A4) % equals "4"
```

```
system_order =
    4
```

```
M = ctrb(A4, B4);
rank(A4)
```

```
ans =
    3
```

```
rank_of_M = rank(M) % equals "4"
```

```
rank_of_M =
    4
```

```
N = obsv(A4, C4);
rank_of_N = rank(N) % equals "4"
```

```
rank_of_N =
    4
```

So the controllability matrix has the rank of 4, while the rank of $A_4 = 3$. This is due to the assumption that all states are available for feedback which matlab doesn't take into account.

With our assumptions we conclude that the system is fully controllable.

The same goes for observability (system is observable).

b)

```
damping = 0.707;
Ts=2;
OS=5;
wn=4/(damping*Ts)
```

```
wn =
    2.8289
```

Natural frequency can be approximated using damping coefficient and settling time.

The equation used is considered "standard" approximation, but in the more complex cases a different formula might have to be implemented to account for non linearities in the system.

```
% Calculate desired pole locations (which are the 2 dominant poles of the
% system)
p1 = -damping * wn + 1i * wn * sqrt(1 - damping^2);
p2 = -damping * wn - 1i * wn * sqrt(1 - damping^2);
% For a fourth-order system, repeat the steps for the second set of poles
p3 = 10*p1; %p3&p4 have to be at least 10 times bigger than the dominant poles
p4 = 10*p2;
% Define desired pole locations
desired_poles = [p1; p2; p3; p4];
% Calculate control gains
K = acker(A4, B4, desired_poles);
% Apply state feedback controller
sys_with_controller = ss(A4 - B4*K, B4, C4, D4);
% Verify the closed-loop system poles
closed_loop_poles=eig(A4 - B4*K)
```

```
closed_loop_poles = 4x1 complex
    -20 +    20.0061i
    -20 -    20.0061i
     -2 +     2.0006i
     -2 -     2.0006i
```

```
eig(A4)
```

```
ans = 4x1 complex
     0 +    0i
-10.077 +    0i
 -5.5998 +   86.204i
 -5.5998 -   86.204i
```

```
disp("Control Gains (K):");
```

Control Gains (K):

```
disp(K);
```

```
    23.094    -79.155    -0.37057     0.85382
```

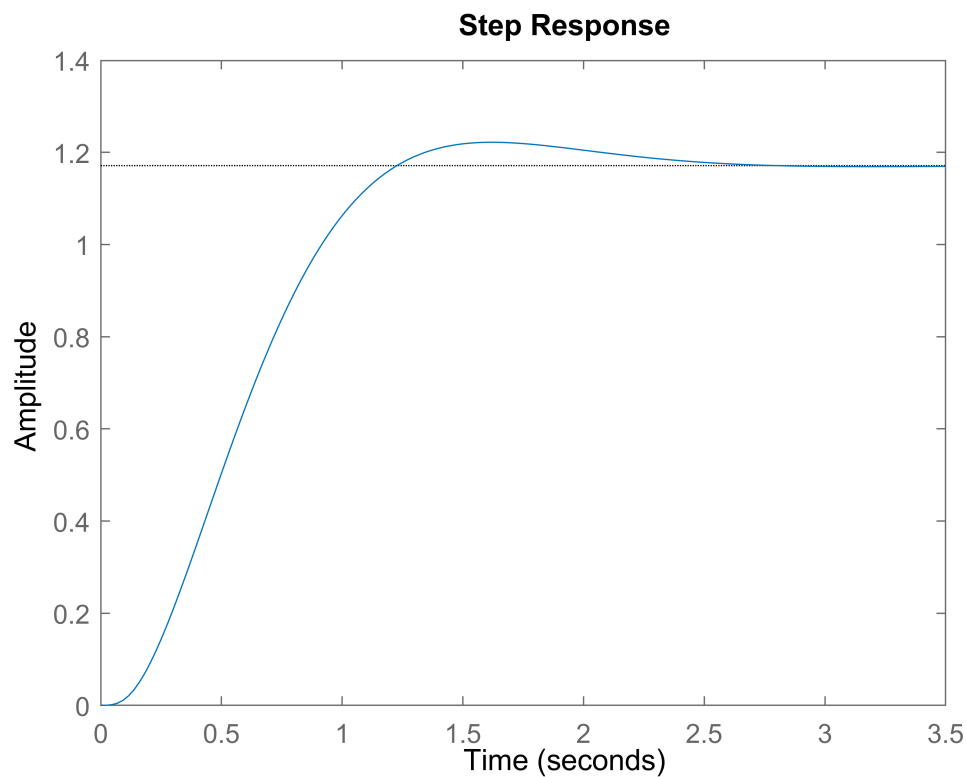
```
disp("Closed-loop System Poles:");
```

Closed-loop System Poles:

```
disp(closed_loop_poles);
```

```
    -20 +    20.0061i
    -20 -    20.0061i
     -2 +     2.0006i
     -2 -     2.0006i
```

```
step(sys_with_controller)
```



```
stepinfo(sys_with_controller)
```

```
ans = struct with fields:
    RiseTime: 0.75955
    TransientTime: 2.1571
    SettlingTime: 2.1571
    SettlingMin: 1.0561
    SettlingMax: 1.2218
    Overshoot: 4.3229
    Undershoot: 0
    Peak: 1.2218
    PeakTime: 1.6118
```

The overshoot is within the specifications, but the settling time is slightly over 2s. We will now adjust the gains to meet the required criteria.

```
%new gain matrix
```

```
K_new=[14.0939679717973    -79.1554023636424    -0.370567971797259     0.853815822485086]
```

```
K_new = 1x4
    14.094    -79.155    -0.37057     0.85382
```

```
% Apply state feedback controller
```

```
sys_with_controller_new = ss(A4 - B4*K_new, B4, C4, D4);
```

```
% Verify the closed-loop system poles
```

```
closed_loop_poles_new=eig(A4 - B4*K_new)
```

```
closed_loop_poles_new = 4x1 complex
    -15.281 +    22.353i
    -15.281 -    22.353i
```

```
-2.2192 + 1.9518i
-2.2192 - 1.9518i
```

```
eig(A4)
```

```
ans = 4x1 complex
      0 + 0i
 -10.077 + 0i
 -5.5998 + 86.204i
 -5.5998 - 86.204i
```

```
disp("Control Gains (K):");
```

```
Control Gains (K):
```

```
disp(K);
```

```
23.094 -79.155 -0.37057 0.85382
```

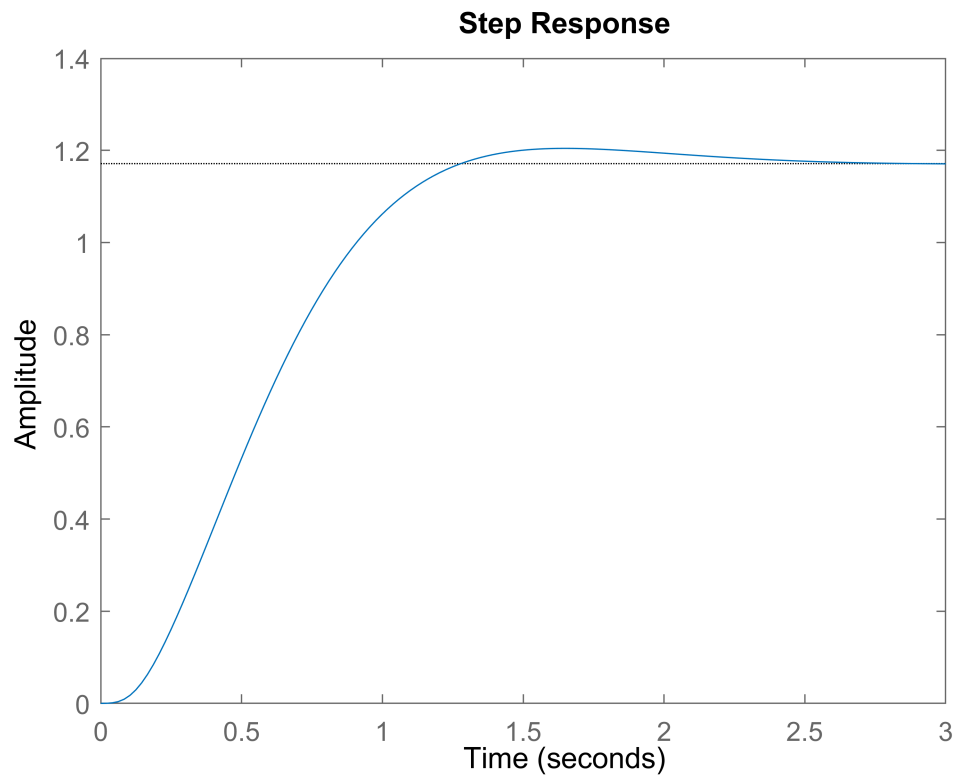
```
disp("Closed-loop System Poles:");
```

```
Closed-loop System Poles:
```

```
disp(closed_loop_poles_new);
```

```
-15.281 + 22.353i
-15.281 - 22.353i
-2.2192 + 1.9518i
-2.2192 - 1.9518i
```

```
step(sys_with_controller_new)
```



```
stepinfo(sys_with_controller_new)
```

```
ans = struct with fields:
    RiseTime: 0.7717
    TransientTime: 1.9815
    SettlingTime: 1.9815
    SettlingMin: 1.0597
    SettlingMax: 1.2043
    Overshoot: 2.8217
    Undershoot: 0
    Peak: 1.2043
    PeakTime: 1.6394
```

After some tuning, we have adjusted the controller such that the system now has Settling time of 1.98s and Overshoot of 2.82%.

```
K_new
```

```
K_new = 1x4
    14.094    -79.155    -0.37057     0.85382
```

This is the gain vector of the tuned controller.

c)

```
Ae = [A4 zeros(size(A4(:,1))); C4 zeros(size(C4(:,1)))]; % Ae = [A4 0; C4 0]
Be = [B4; zeros(size(B4(1,:)))]; % Be = [B4; 0]
Ce = [C4 zeros(size(C4(:,1)))]; % Ce = [C4 0]
De = [0];
```

```
%now system is 5x5 due to integral element
system_order = length(Ae)
```

```
system_order =
    5
```

```
M = ctrb(Ae, Be);
rank_of_M = rank(M)
```

```
rank_of_M =
    5
```

```
N = obsv(Ae, Ce);
rank_of_N = rank(N)
```

```
rank_of_N =
    4
```

```
[num2,den2] = ord2(wn, damping);
dominant = roots(den2); % dominant complex pole pair

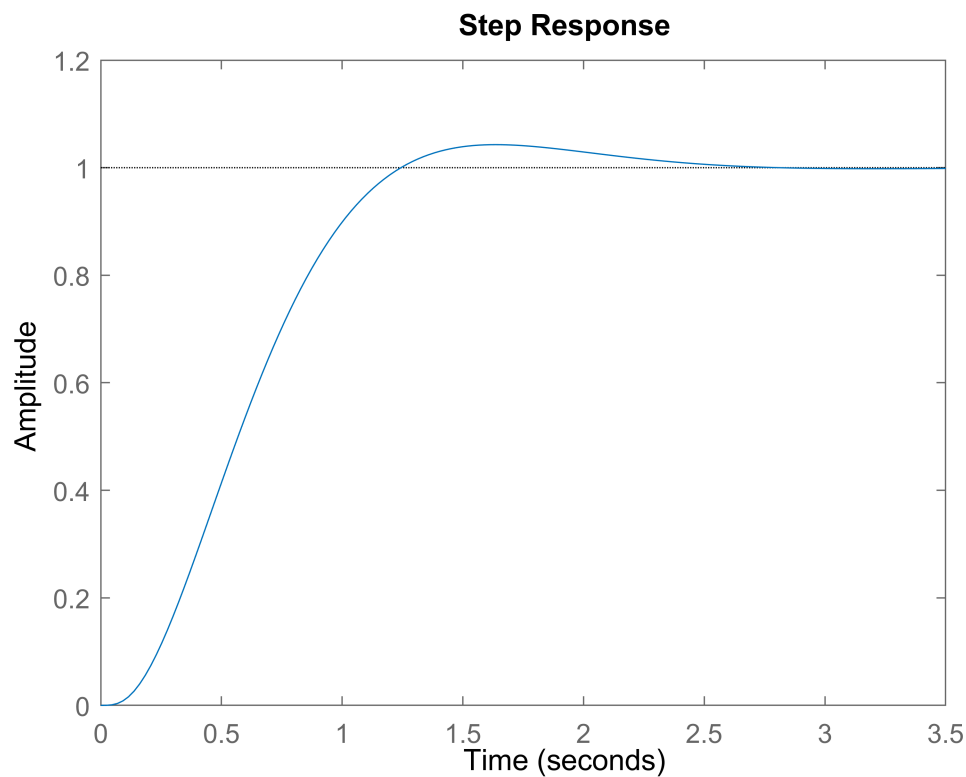
desiredpoles_ext = [dominant' 10*conj(dominant(1)) 20*conj(dominant(2)) 40*min(dominant)];

K_e = acker(Ae, Be, desiredpoles_ext)
```

```
Warning: Pole locations are more than 10% in error.
```

```
K_e = 1x5
    110.97    -3.3451    11.755     76.685    136.61
```

```
Asf_ext = Ae - Be * K_e;
[numsf_ext, densf_ext] = ss2tf(Asf_ext, [0; 0; 0; 0; -1], Ce, De);
SYS_ext = tf(numsf_ext, densf_ext);
step(SYS_ext)
```

```
stepinfo(SYS_ext)
```

```
ans = struct with fields:
    RiseTime: 0.7661
    TransientTime: 2.1702
    SettlingTime: 2.1702
    SettlingMin: 0.90637
    SettlingMax: 1.0429
    Overshoot: 4.2882
    Undershoot: 0
    Peak: 1.0429
    PeakTime: 1.6348
```

Once again the settling time has to be adjusted

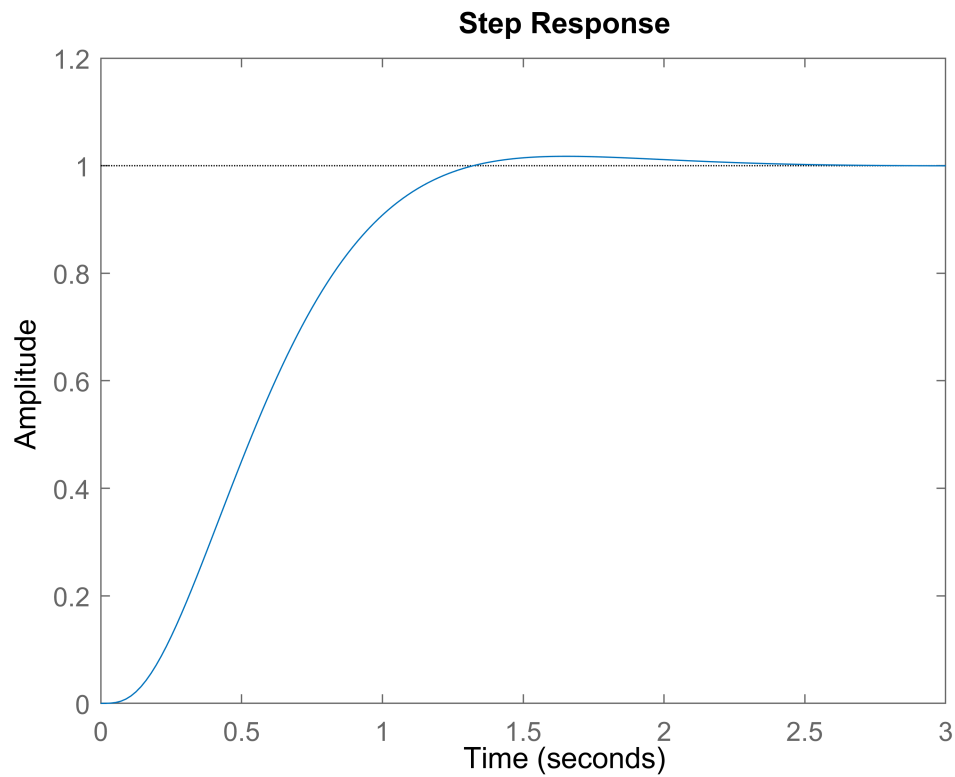
```
%new gain matrix
```

```
K_e_new=[110.968452424576    -5.34514841462968...
          8.7549475754236     76.6851540744224     136.610531597614]
```

```
K_e_new = 1x5
    110.97    -5.3451     8.7549    76.685    136.61
```

```
% Apply state feedback controller
```

```
Asf_ext = Ae - Be * K_e_new;
[numsf_ext, densf_ext] = ss2tf(Asf_ext, [0; 0; 0; 0; -1], Ce, De);
SYS_ext = tf(numsf_ext, densf_ext);
step(SYS_ext)
```



```
stepinfo(SYS_ext)
```

```
ans = struct with fields:
    RiseTime: 0.75667
    TransientTime: 1.2096
    SettlingTime: 1.2096
    SettlingMin: 0.90009
    SettlingMax: 1.0175
    Overshoot: 1.7462
    Undershoot: 0
    Peak: 1.0175
    PeakTime: 1.6455
```

After adjusting the gain matrix shown above, we have obtained the system with integral element which satisfies all 4 conditions imposed ($\text{ess}=0$, $T_s < 2$ & $\text{OS} < 5\%$).

TASK6

```
%Gains of the PID controller from the table
```

```
Kp = 2.7*wn^3;
Ki = wn^4;
Kd = 3.4*wn^2 - 12;
num_cont=[Kd Kp Ki]
```

```
num_cont = 1×3
    15.208    61.122    64.039
```

```
den_cont=[1 0]
```

```
den_cont = 1×2  
          1    0
```

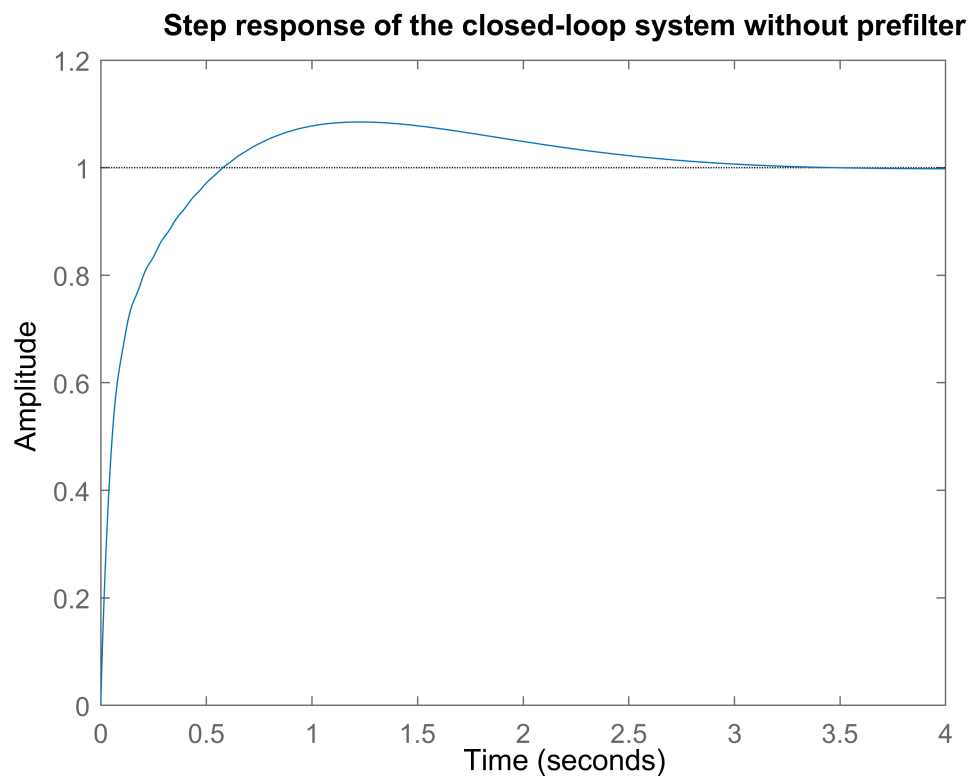
```
Gc = tf(num_cont,den_cont)
```

```
Gc =
```

```
    15.21 s^2 + 61.12 s + 64.04  
-----  
                s
```

Continuous-time transfer function.

```
T1 = feedback(Gc*G4s,1);  
figure();  
step(T1);  
title('Step response of the closed-loop system without prefilter');
```



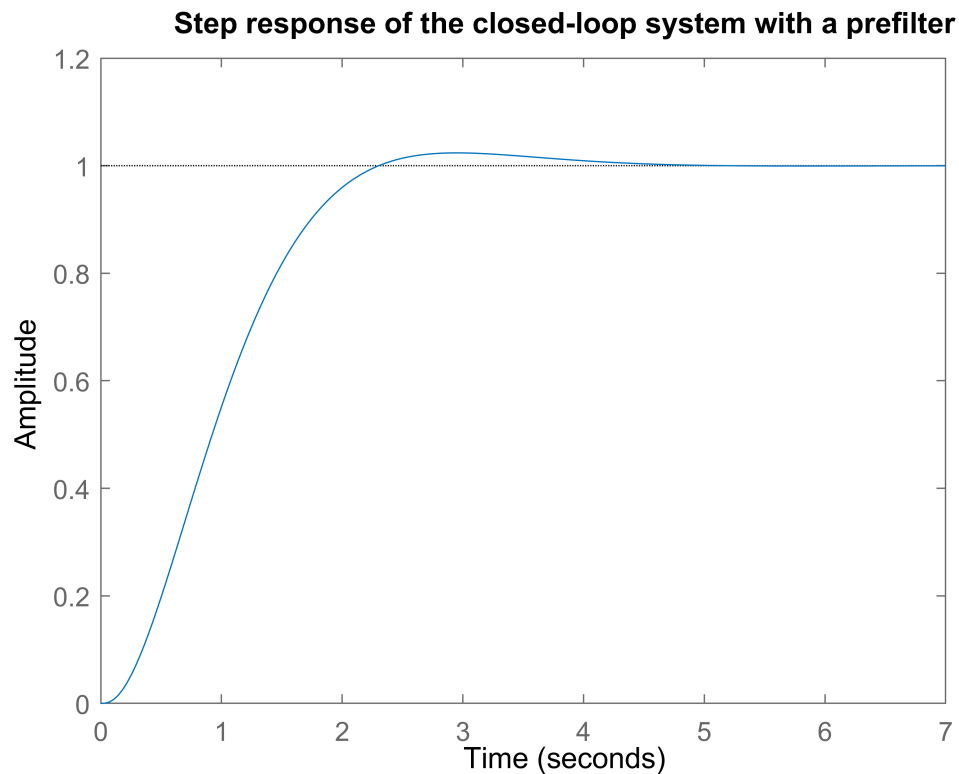
The introduction of the PID controller using Ziegler-Nichols method allowed us to obtain a step response with 0 steady state error, but unsatisfied remaining conditions.

We will now add a prefilter which will allow the PID controller to properly satisfy all the criteria.

```
%creating prefilter  
num_pre=Ki;  
den_pre=[Kd Kp Ki]
```

```
den_pre = 1×3  
        15.208    61.122    64.039
```

```
Gp=tf(num_pre, den_pre);  
T1 = feedback(Gc*G4s,1);  
T = T1*Gp;  
figure()  
step(T);  
title('Step response of the closed-loop system with a prefilter')
```



```
stepinfo(T)
```

```
ans = struct with fields:  
    RiseTime: 1.3962  
    TransientTime: 3.352  
    SettlingTime: 3.352  
    SettlingMin: 0.90367  
    SettlingMax: 1.0238  
    Overshoot: 2.3835  
    Undershoot: 0  
    Peak: 1.0238  
    PeakTime: 2.936
```

```
% Tuning
```

Now we proceed with PID tuning using a random permutation test bench, iterating through random values of K_p , K_d and K_i in a given range until all conditions are satisfied or code iteration limit.

```

% condition = false;
% iterator=0;
% while condition==false
%
% %Gains of the PID controller from the table
% Kp = 2.7*wn^3;
% Kp=Kp+0.5*Kp*(0.5-rand(1));
% Ki = wn^4;
% Ki=Ki+0.5*Ki*(0.5-rand(1));
% Kd = 3.4*wn^2 - 12;
% Kd=Kd+0.5*Kd*(0.5-rand(1));
% num_cont=[Kd Kp Ki];
% den_cont=[1 0];
% Gc = tf(num_cont,den_cont);
% T1 = feedback(Gc*G4s,1);
%
%
% num_pre=[Ki];
% den_pre=[Kd Kp Ki];
% Gp=tf(num_pre, den_pre);
% T1 = feedback(Gc*G4s,1);
% T = T1*Gp;
%
% data=stepinfo(T);
% Ov=data.Overshoot;
% ST=data.SettlingTime;
%
% if Ov<=5 && ST <=2
%     condition=true;
% end
%
% iterator=iterator+1;
%
% if iterator==1000
%     condition=true;
% end
% end

```

From code above we obtain:

Kp=63.255

Kp =
63.255

Ki=74.131

Ki =
74.131

Kd=12.129

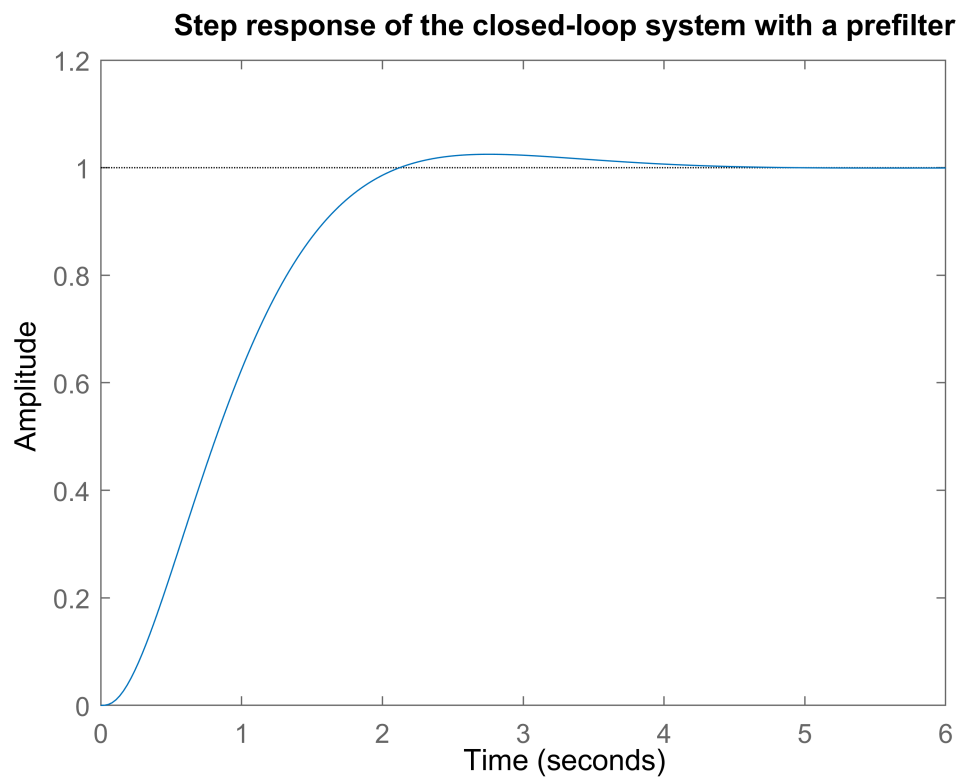
Kd =
12.129

Creating a new controller using these values:

```
num_pre=[Ki];  
den_pre=[Kd Kp Ki]
```

```
den_pre = 1×3  
12.129    63.255    74.131
```

```
Gp=tf(num_pre, den_pre);  
T1 = feedback(Gc*G4s,1);  
T = T1*Gp;  
figure()  
step(T);  
title('Step response of the closed-loop system with a prefilter')
```



```
stepinfo(T)
```

```
ans = struct with fields:  
    RiseTime: 1.29  
    TransientTime: 3.2079  
    SettlingTime: 3.2079  
    SettlingMin: 0.90523  
    SettlingMax: 1.025  
    Overshoot: 2.4957  
    Undershoot: 0  
    Peak: 1.025  
    PeakTime: 2.7603
```

After adjusting the PID coefficients our system meets all the conditions. In order to get it to function properly we had to create an initial PID controller and equip it with appropriate prefilter resulting in a correct solution.